

# Solution of the truss problem

First of all the data needs to be declared. These include

- the number of nodes
- the number of elements
- the coordinates of the nodes
- the connectivity of the members
- the element stiffness
- the nodal forces vector
- the boundary conditions

To have consistent units the Young modulus is expressed in [kN/m<sup>2</sup>], the area in [m<sup>2</sup>] and the length in [m]. This means that the external forces need to be expressed in [kN].

The obtained displacement will be expressed in [m] and the reaction forces are also expressed in [kN].

The following code declares the data for the problem

```
In [1]: import numpy as np

A = 4000 * 1e-6 # m^2
E = 70 * 1e6 # kPa

EA = A * E

L_1 = 6 # m
L_2 = 4 # m
H = 8 # m

# The nodes are numbered from the bottom to the top, from the left to the right

node_1 = np.array([0, 0])
node_2 = np.array([L_1 + L_2, 0])
node_3 = np.array([0, H])
node_4 = np.array([L_1, H])

coordinates = np.vstack((node_1, node_2, node_3, node_4))

connectivity_table = np.array([[1, 3],
                               [3, 4],
                               [1, 4],
                               [2, 3],
                               [2, 4]]) - 1 # -1 because of python convention

n_nodes = coordinates.shape[0]
```

```

n_elements = connectivity_table.shape[0]

n_dofs = n_nodes * 2

# Dictionary containing information on the force value at solicited nodes
dict_forces_1 = {3: np.array([0, - 400]),
                 4: np.array([800, - 400])}

f = np.zeros(n_dofs)

for node, force_node in dict_forces_1.items():
    node_id = node - 1
    f[2*node_id:2*node_id+2] = force_node

# Node 1, 2 fixes the x, y displacement. Node 3 fixes the x displacement
dofs_bcs = [0, 1, 2, 3, 4]

```

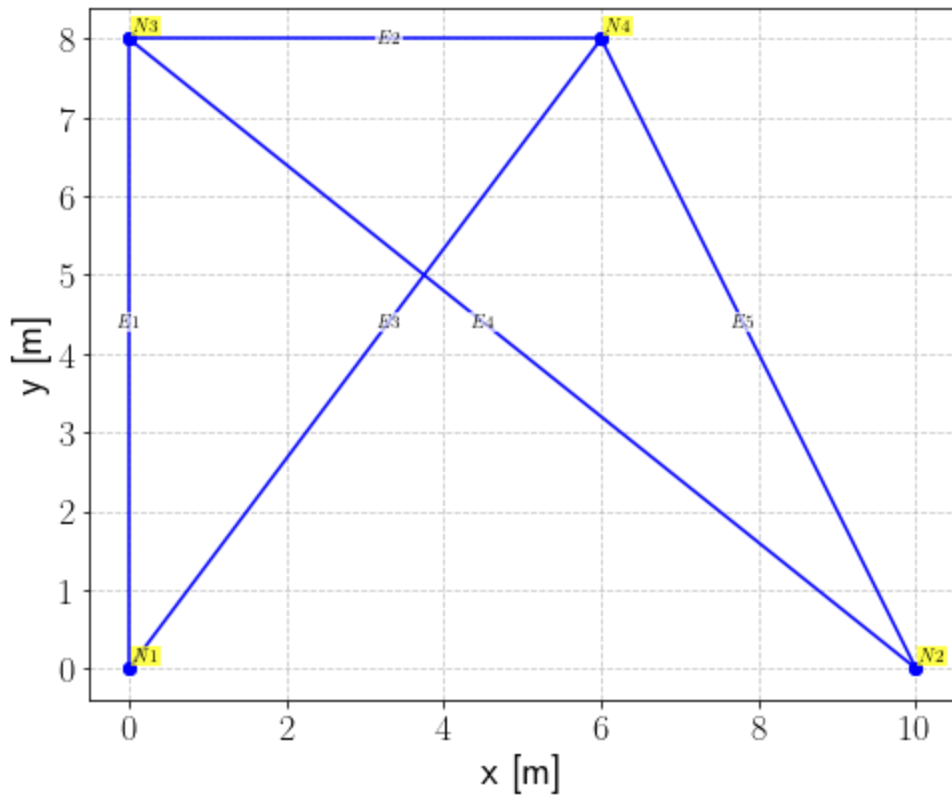
Then the second part consists in assembling the stiffness matrix. First of all we verify that the mesh is correct by plotting the different elements

```

In [2]: from matplotlib import pyplot as plt
        from src.postprocessing.plot_mesh import plot_truss_structure

        ax = plot_truss_structure(coordinates, connectivity_table,
                                   show_element_labels=True,
                                   show_node_labels=True,
                                   color='blue',
                                   linestyle='o-',
                                   xlabel='x [m]',
                                   ylabel='y [m]',)

```



Once the mesh has been verified, we can proceed to the construction of the stiffness matrix

```
In [3]: from src.fem.assemble_stiffness import assemble_stiffness_truss_2d
from src.fem.solve_system import solve_system_homogeneous_bcs

K, _, _ = assemble_stiffness_truss_2d(coordinates, connectivity_table, EA)
q_all, reactions = solve_system_homogeneous_bcs(K, f, dofs_bcs)

q_all_mm = [round(v, 4) for v in 1e3*q_all]
print(f"Displacement at these nodes [mm]:\n {q_all_mm}\n")
print(f"Reactions at bcs in [kN]:\n {reactions}\n")
```

Displacement at these nodes [mm]:

```
[0.0, 0.0, 0.0, 0.0, 0.0, -9.1886, 12.8365, -9.5844]
```

Reactions at bcs in [kN]:

```
[ -0.57760745  320.82925204 -298.3858275   479.17074796 -501.03656505]
```

```
In [4]: # Plot the obtained solution by amplifying the displacements
from src.postprocessing.plot_mesh import plot_truss_structure

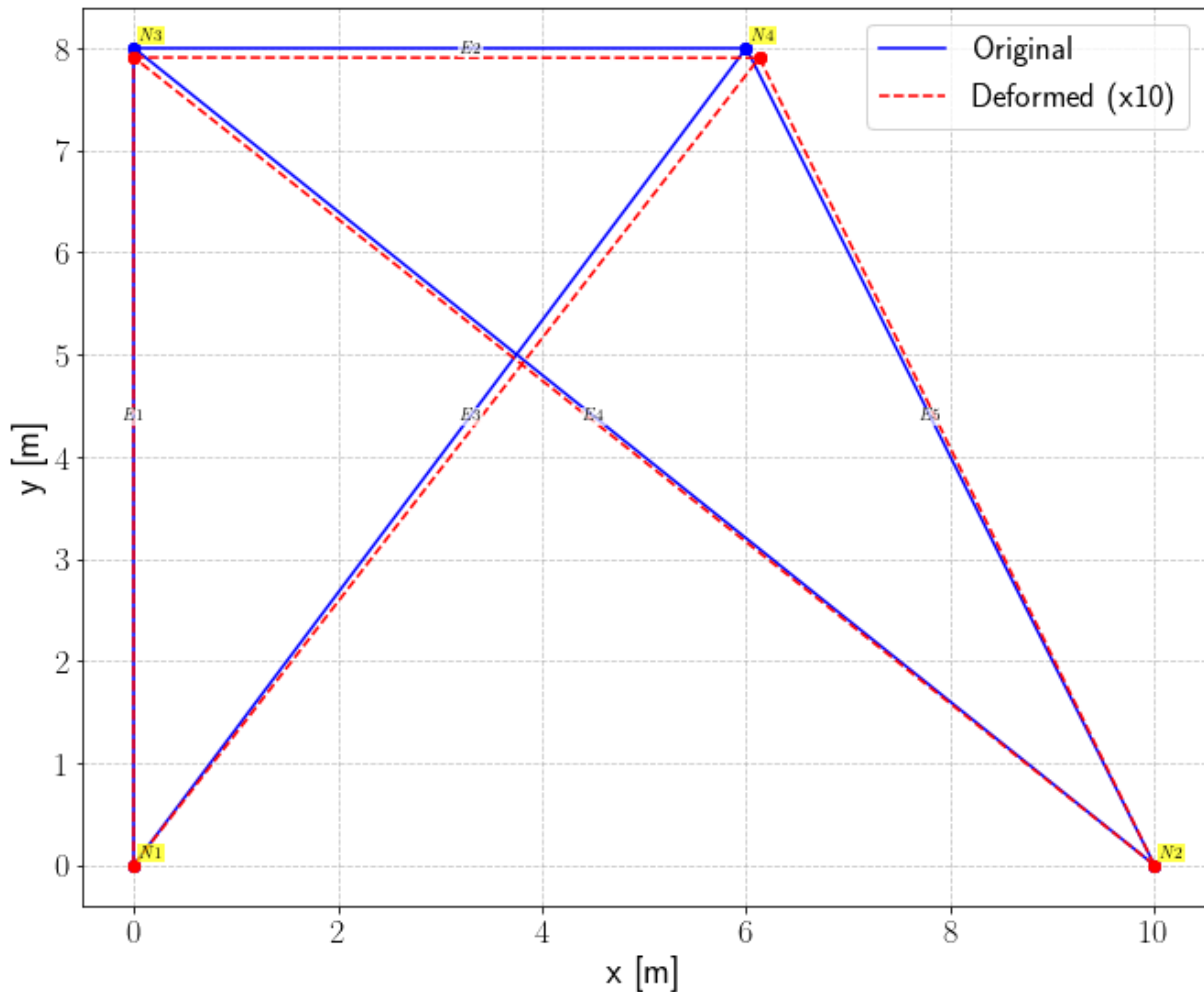
fig, ax = plt.subplots(figsize=(10, 8))
ax = plot_truss_structure(coordinates, connectivity_table, ax=ax,
    show_element_labels=True,
    show_node_labels=True,
    color='blue',
    linestyle='o-',
    label='Original',
    xlabel='x [m]',
```

```

        ylabel='y [m]',)

scale = 10
deformed_coordinates = coordinates + scale * q_all.reshape(-1, 2)
ax = plot_truss_structure(deformed_coordinates, connectivity_table, ax=ax, c=
        linestyle='o--',
        show_element_labels=False,
        show_node_labels=False,
        label=f'Deformed (x{scale})')
plt.tight_layout()
plt.show()

```



Then we can reconstruct the displacements and reaction forces at each node and find the axial force in each element

```

In [5]: from src.fem.compute_strains_forces import compute_strains_forces_truss_2d

f_all = K @ q_all

axial_strains, axial_forces = compute_strains_forces_truss_2d(coordinates, \
        connectivity_table, q_all)

for ii in range(n_elements):
    print(f"Axial force in element {ii+1} [MPa]: \n {axial_forces[ii]:.3f}")

```

```
Axial force in element 1 [MPa]:  
-321.599  
Axial force in element 2 [MPa]:  
599.037  
Axial force in element 3 [MPa]:  
0.963  
Axial force in element 4 [MPa]:  
-125.502  
Axial force in element 5 [MPa]:  
-448.075
```

The axial forces depend on the physical parameters of each truss (structure is statically indetermined). We can verify it by changing the parameters and compute the forces again

```
In [ ]: from scipy.sparse.linalg import norm  
  
E_2 = 80 * 1e6 # kPa  
A_2_vec = np.array([4000, 3000, 5000, 6000, 7000]) * 1e-6  
EA_2 = A_2_vec * E_2  
  
K_2, _, _ = assemble_stiffness_truss_2d(coordinates, connectivity_table, EA_2)  
q_all_K2, reactions_K2 = solve_system_homogeneous_bcs(K_2, f, dofs_bcs)  
  
print(f'Reactions for K2: \n {reactions_K2}')  
axial_strains_K2, axial_forces_K2 = compute_strains_forces_truss_2d(coordinates,  
                                                                    connectivity_table, q_all_K2)  
  
for ii in range(n_elements):  
    print(f"Difference local force in element {ii+1}: \n {axial_forces_K2[ii]} -
```

```
Reactions for K2:  
[ -82.30238595  183.1589079  -388.74897917  616.8410921  -328.94863488]  
Difference local force in element 1:  
-28.7 [kN]  
Difference local force in element 2:  
136.2 [kN]  
Difference local force in element 3:  
-136.2 [kN]  
Difference local force in element 4:  
45.9 [kN]  
Difference local force in element 5:  
121.8 [kN]
```

```
In [ ]: # Second load case  
dict_forces_2 = {4: np.array([1500, 0])}  
f_2 = np.zeros(n_dofs)  
  
for node, force_node in dict_forces_2.items():  
    node_id = node - 1  
    f_2[2*node_id:2*node_id+2] = force_node  
  
q_all_f2, reactions_f2 = solve_system_homogeneous_bcs(K, f_2, dofs_bcs)
```

```

print(f"Displacement [mm]:\n {q_all_f2*1e3}\n")
print(f"Reactions at bcs in [kN]:\n {reactions_f2}\n")

f_all_f2 = K @ q_all_f2

axial_strains_f2, axial_forces_f2 = compute_strains_forces_truss_2d(coordinates,
                                                                    connectivity_table, q_all_f2)

for ii in range(n_elements):
    print(f"Local axial force element {ii+1} in [kN]: \n {axial_forces_f2[ii]}")

```

In order to find the optimal parameters to minimize mass and respect the safety requirements, we compute the area so that the stress is the maximum acceptable.

```

In [9]: sigma_max_MPa = 200 # [MPa]
        safety_factor = 2

        axial_sigma_kPa = E*axial_strains
        axial_sigma_MPa = axial_sigma_kPa/1e3
        print(f'Axial stress in [MPa]: \n {axial_sigma_MPa}')
        sigma_max_security = sigma_max_MPa/2
        sigma_ideal = sigma_max_security

        A_opt = abs(axial_sigma_MPa)*A/sigma_ideal
        assert np.isclose(abs(axial_sigma_kPa*A), abs(axial_forces)).all()
        print(f'Ration cross section')
        for ii in range(n_elements):
            print(f'Ratio element {ii+1}: {A_opt[ii]/A}')

```

```

Axial stress in [MPa]:
[ -80.39984883  149.75933023   0.24066977 -31.37555076 -112.01866046]
Ration cross section
Ratio element 1: 0.8039984882526215
Ratio element 2: 1.4975933023045926
Ratio element 3: 0.0024066976954077554
Ratio element 4: 0.3137555076107792
Ratio element 5: 1.1201866046091846

```

This is however not sufficient as the different areas will induce different internal stresses. We can verify this by computing again the stresses

```

In [10]: EA_opt = E * A_opt
        K_opt, _, _ = assemble_stiffness_truss_2d(coordinates, connectivity_table, E)

        q_all_opt, reactions_opt = solve_system_homogeneous_bcs(K_opt, f, dofs_bcs)

        axial_strains_opt, axial_forces_opt = compute_strains_forces_truss_2d(coordinates,
                                                                                connectivity_table, q_all_opt)

        axial_stresses_opt_MPa = E*axial_strains_opt/1e3
        for ii in range(n_elements):
            print(f"Local axial stress element {ii+1} in [MPa]: \n {axial_stresses_opt_MPa[ii]}")

```

Local axial stress element 1 in [MPa]:  
-113.6  
Local axial stress element 2 in [MPa]:  
100.2  
Local axial stress element 3 in [MPa]:  
-19.7  
Local axial stress element 4 in [MPa]:  
-44.3  
Local axial stress element 5 in [MPa]:  
-99.8

This solution needs to be obtained via an iterative process, either via a constrained optimization algorithm, either via an heuristic reasoning that converges iteratively to an acceptable solution.