

- It is required to design an air-cooled condenser – refrigerant will condense in the tubes at a temperature of 38°C with a film heat transfer coefficient of 525 W/m<sup>2</sup>K. Incoming air at 23°C is available. It is proposed to use a face air velocity of 3.1 m/s at which velocity the convection heat transfer coefficient to the fin sheet will be 23 W/m<sup>2</sup>K. It has been recommended that the unit have fin sheets of dimensions 0.85 m high by 0.42 m deep. The refrigerant carrying tubes have an inside diameter of 18mm and an outside diameter of 21.5 mm. The tubes are arranged on triangular pitch with centre to centre spacing of 53 mm. The fins are 0.4 mm thick aluminium ( $k = 200 \text{ W/mK}$ ) and are spaced at 6.35 mm centres. In each tube sheet there are 120 tube passes. The required heat transfer is 28.6 kW. Determine the length of the condenser, and the number of fin sheets required.

(Note: -  $c = 1010 \text{ J/kgK}$  for air)

- A continuous carton freezer for beef operates at -35°C air temperature. Cartons enter at 10°C and leave at a mass average temperature of -20°C. Each day 2800 cartons, each containing 27.2kg of meat are frozen. The thermal properties of the beef are:- frozen specific heat capacity  $1.9 \times 10^6 \text{ J/m}^3\text{K}$ , unfrozen specific heat capacity  $3.6 \times 10^6 \text{ J/m}^3\text{K}$ , the latent heat of freezing is  $207 \times 10^3 \text{ J/kg}$ , the temperature at which freezing commences -1°C.

The freezer is totally enclosed in a large multi-storey cold store which operates at -16°C. All the walls, ceiling and floor are constructed of 150mm thick sandwich panel made of polyurethane ( $k = 0.026 \text{ W/mK}$ ). The dimensions of the freezer are 4m x 8m x 20m. Loading of cartons occurs for a 10 hour period every day. During this time an unprotected doorway 2m x 1m is continually open, and a velocity through it of 1.05 m/s occurs. Also during this time 2 men are present, and they use 550 W lighting.

There are eight similar evaporators, each having a face area of 2.2m x 1.55m, and a velocity leaving the face of 3.2 m/s. the pressure drop the fans work against has been estimated at 54mm water gauge and the fan efficiency is 0.63. The fan motor efficiency has been estimated at 0.93. Each evaporator is defrosted 1 hour per week. Estimate the 24 hour mean heat load.

- For a cool store with a heat load of 15 kW, and mean temp difference of 12.3 K, it has been proposed to use a finned air-cooled condenser. The fin sheets are 0.85 m high and 0.4 m deep. The length of the heat exchanger is 1.18m. The fins are spaced 4.5 mm apart, and are 0.5 mm thick so there are 225 fins in the length of the heat exchanger. The tubes are arranged on square pitch with 65mm pipe centres, there being  $(14 \times 6) = 84$  tubes, giving a total tube length of 99.1m. The condensation film coefficient is 450 W/m<sup>2</sup>K and the airside coefficient 18 W/m<sup>2</sup>K. the pipes are 20mm outside diameter, 17mm inside diameter, and both they and the fins are aluminium (thermal conductivity = 200 W/mK). Is there sufficient heat transfer capability?
- Peas are frozen in fluidised bed freezers in which the surface heat transfer coefficient is typically 200 W/m<sup>2</sup>K. The peas can be treated as spheres of diameter 10 mm. The peas enter the freezer at 20°C, start to freeze at about -1°C, and must reach a final centre temperature of -18°C. A residence time of 150 s and an air temperature of -36°C has been suggested. Determine whether this residence time is satisfactory. Thermal properties of peas are:- frozen thermal conductivity 1.68 W/mK, unfrozen thermal conductivity 0.49 W/mK, frozen specific heat capacity  $2.1 \times 10^6 \text{ J/m}^3\text{K}$ , unfrozen specific

heat capacity  $3.7 \times 10^6 \text{ J/m}^3\text{K}$ , enthalpy change from  $-1^\circ\text{C}$  to  $-20^\circ\text{C}$   $238 \times 10^6 \text{ J/m}^3$ , density  $1015 \text{ kg/m}^3$ .

5. Fish fillet are packed in cartons for freezing – the cartons have dimensions of  $75\text{mm} \times 295\text{mm} \times 410\text{mm}$ , a wall thickness of  $2.8 \text{ mm}$ , and a wall fiberite thermal conductivity of  $0.062 \text{ W/mK}$ . When frozen the last point to freeze is not the geometric centre because the air space left in the top of the carton reduces heat transfer to that surface. The last point to freeze is  $43\text{mm}$  up from the bottom of the carton. A rule of thumb can be applied here for uneven heat transfer:- freezing time for a  $75\text{mm}$  carton with uneven heat transfer = freezing time for a  $(2 \times 43) = 86\text{mm}$  carton with the surface heat transfer coefficient for the bottom surface = freezing time for a  $(2 \times 32) = 64 \text{ mm}$  thick carton with the heat transfer coefficient is difficult to estimate so the calculation will be carried out for a  $86 \text{ mm}$  thick carton with no air gap between product and surface. The fillets enter the cartons at  $80^\circ\text{C}$  and must be frozen to a final centre temperature of  $-20^\circ\text{C}$  within 9 hours. Air at  $-38^\circ\text{C}$  is to be used. Determine the air velocity required. Thermal properties of the fish are:- frozen thermal conductivity  $1.59 \text{ W/mK}$ , unfrozen thermal conductivity  $0.51 \text{ W/mK}$ , frozen specific heat capacity  $2.1 \times 10^6 \text{ J/m}^3\text{K}$ , unfrozen specific heat capacity  $3.5 \times 10^6 \text{ J/m}^3\text{K}$ , enthalpy change from  $-1^\circ\text{C}$  to  $-10^\circ\text{C}$  is  $228 \times 10^6 \text{ J/m}^3$ , density  $1022 \text{ kg/m}^3$ .

(Note:- in order to calculate the surface heat transfer coefficient from Pham's modification to Plank's equation it is necessary to know  $E$ , which itself is dependent on the surface heat transfer coefficient. Hence assume an air velocity e.g.  $2-5 \text{ m/s}$  and find an initial estimate of  $h_e$  can be calculated from the freezing time equation to see if it corresponds).

6. You are working for a company that produces pre-cooked minced beef chilli con carne. The product is vacuum sealed into plastic pouches ( $200\mu\text{m}$  thick) and cooked to a centre temperature of  $75^\circ\text{C}$ . It is then required that the product be cooled to a centre temperature of  $5^\circ\text{C}$ . Two cooling options are being considered, air blast chilling and immersion chilling. You are required to predict and compare the required cooling times for the two processes. The following data is available for your predictions

The thermal properties for the minced beef are:

Unfrozen specific heat:  $3.44 \text{ kJ/kgK}$

Unfrozen thermal conductivity:  $0.68 \text{ W/mK}$

Density:  $1050 \text{ kg/m}^3$

Dimensions of the packaging are  $235 \times 120 \times 35 \text{ mm}$

The immersion cooler will operate with brine at  $0^\circ\text{C}$  and a heat transfer coefficient to the surface of the package is estimated to be  $150\text{W/m}^2\text{K}$ .

The air blast chiller has an air flow of  $3\text{m/s}$  over the product surface and will operate with an air temperature of  $0^\circ\text{C}$ .

Compared the predicted times and discuss the advantages and disadvantages of the two cooling processes.

7. A 57 kg side of pork is being chilled to a centre temperature of 5°C from an initial temperature of 37°C using air at 2°C and a velocity of 3m/s. The critical area is the deep leg position (hip bone). The measured dimensions in the 3 directions are 115 mm x 220 mm x 385 mm.

Thermal properties of pork are

Unfrozen specific heat = 2800 J/kgK

Density = 1040 kg/m<sup>3</sup>

Unfrozen thermal conductivity = 0.48 W/mK

- a) Calculate the time required for cooling.
- b) Calculate the mass average temperature at the completion of cooling and the average product heat load if 30 sides are to be chilled per batch.