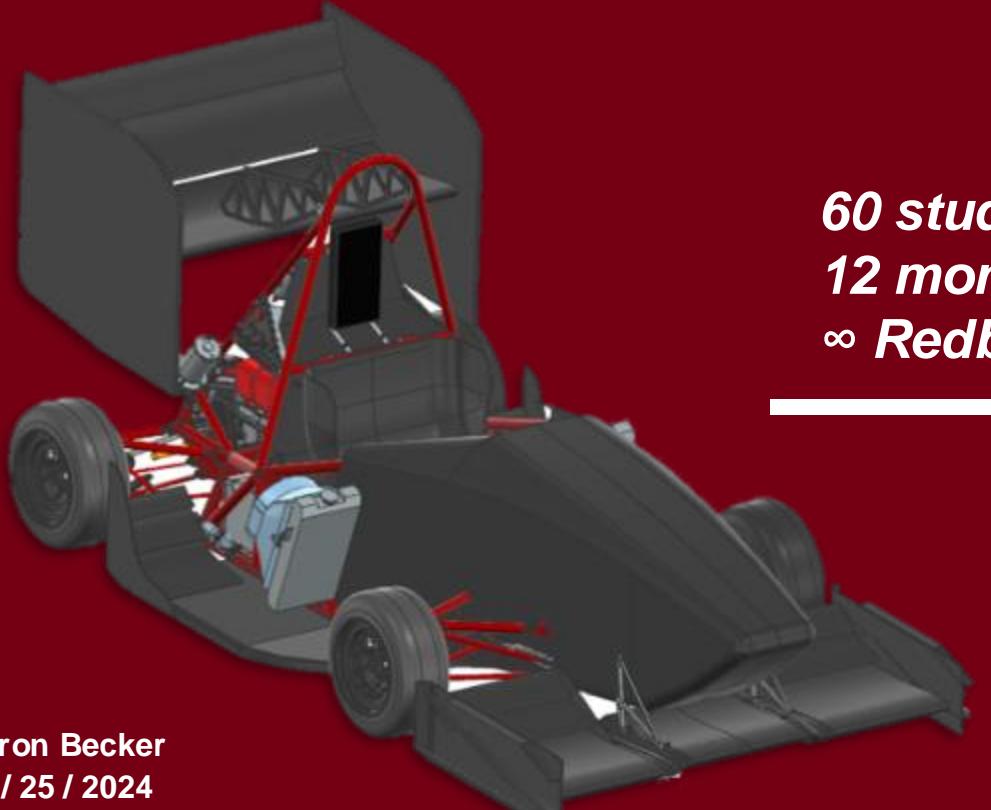


Formula SAE at MIT

MIT students building racecars!



*60 students
12 months
∞ Redbull*



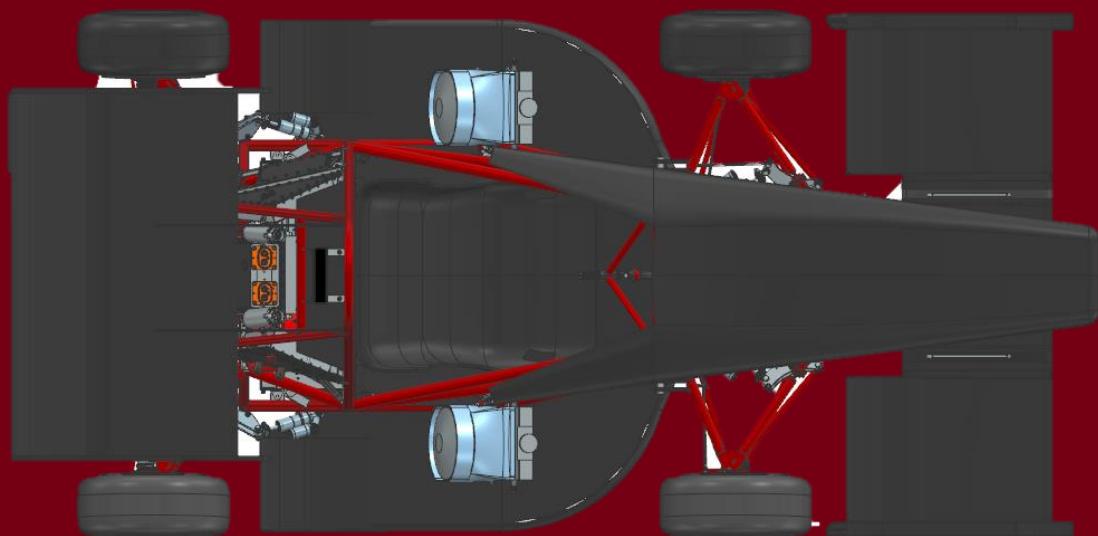


What's FSAE all about?



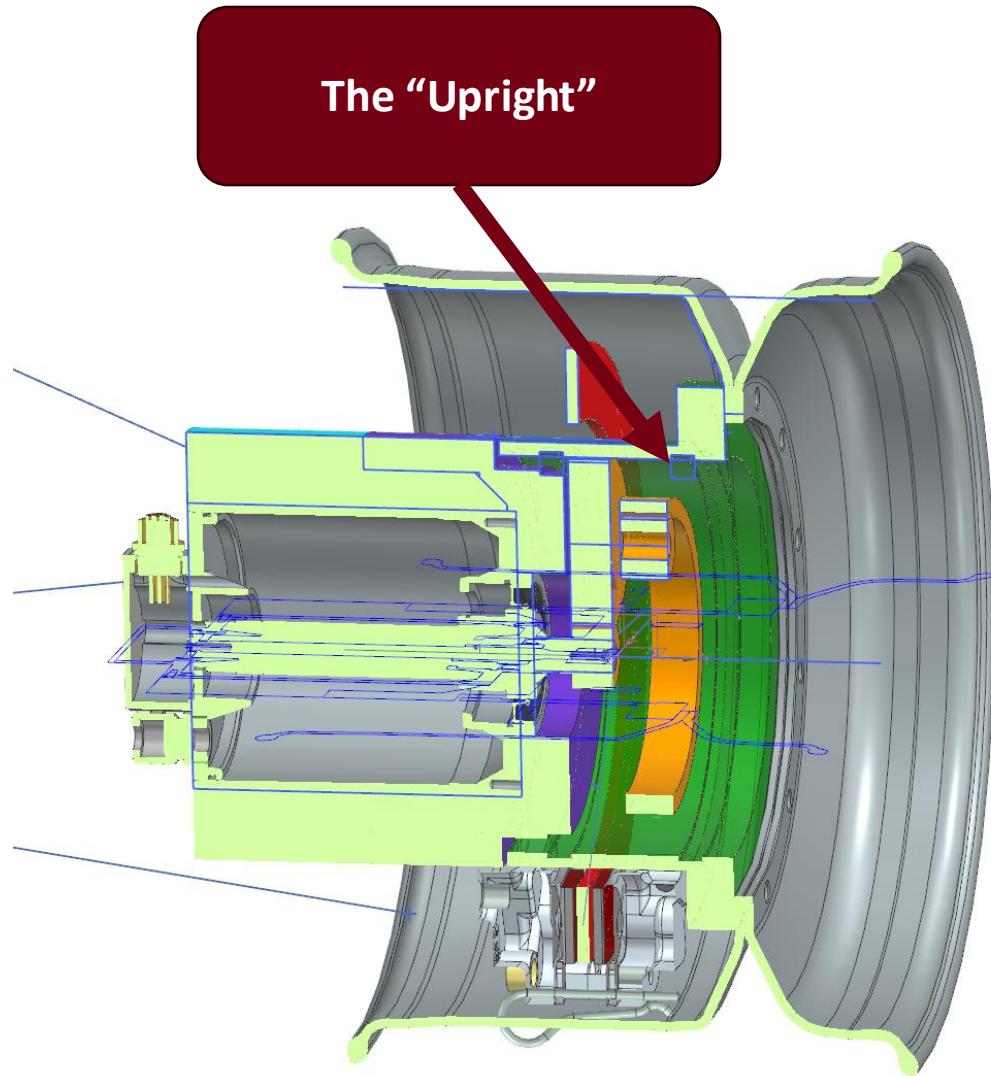
What's FSAE all about?

- Build fast (electric) cars, and race them **safely**
- Learn to be great engineers by following a *design process*
- Have fun!



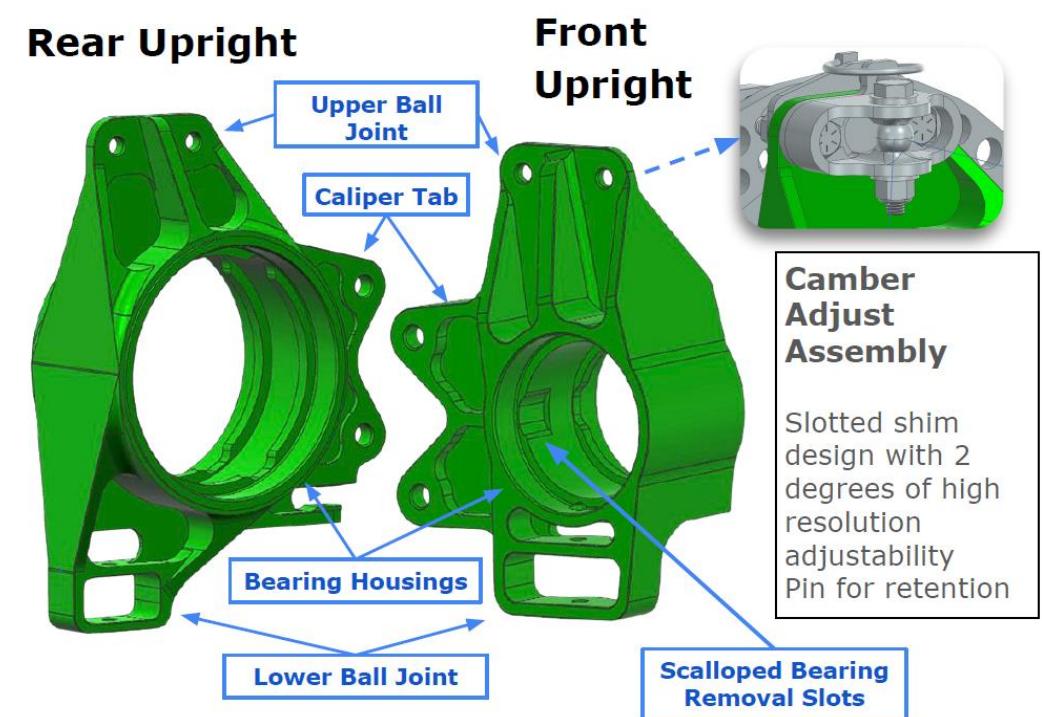
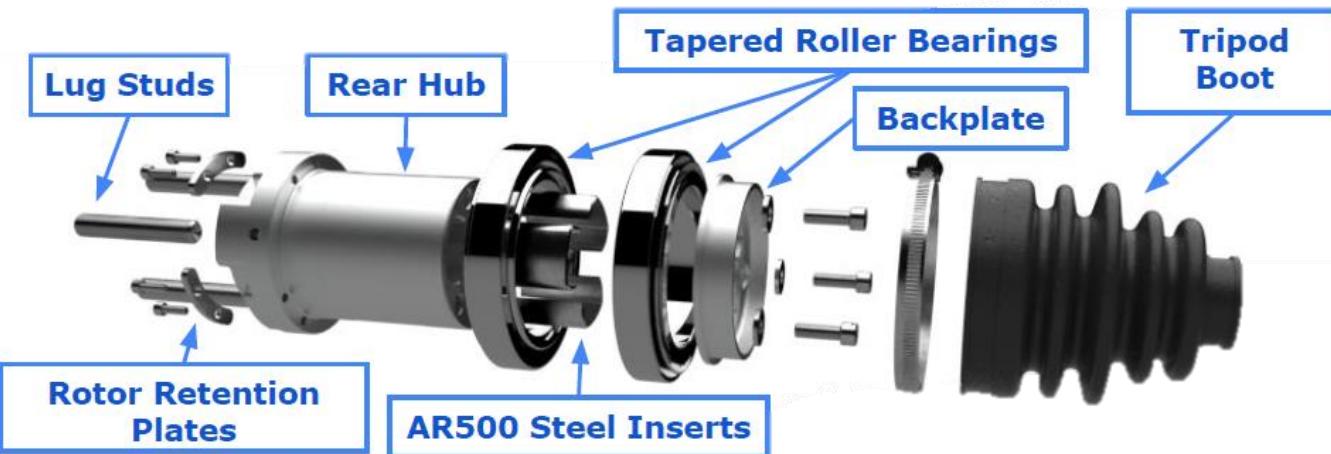
Let's follow a part!

- Explain a year-long “design cycle”
- Talk about some of the fantastic resources at MIT that make it possible!

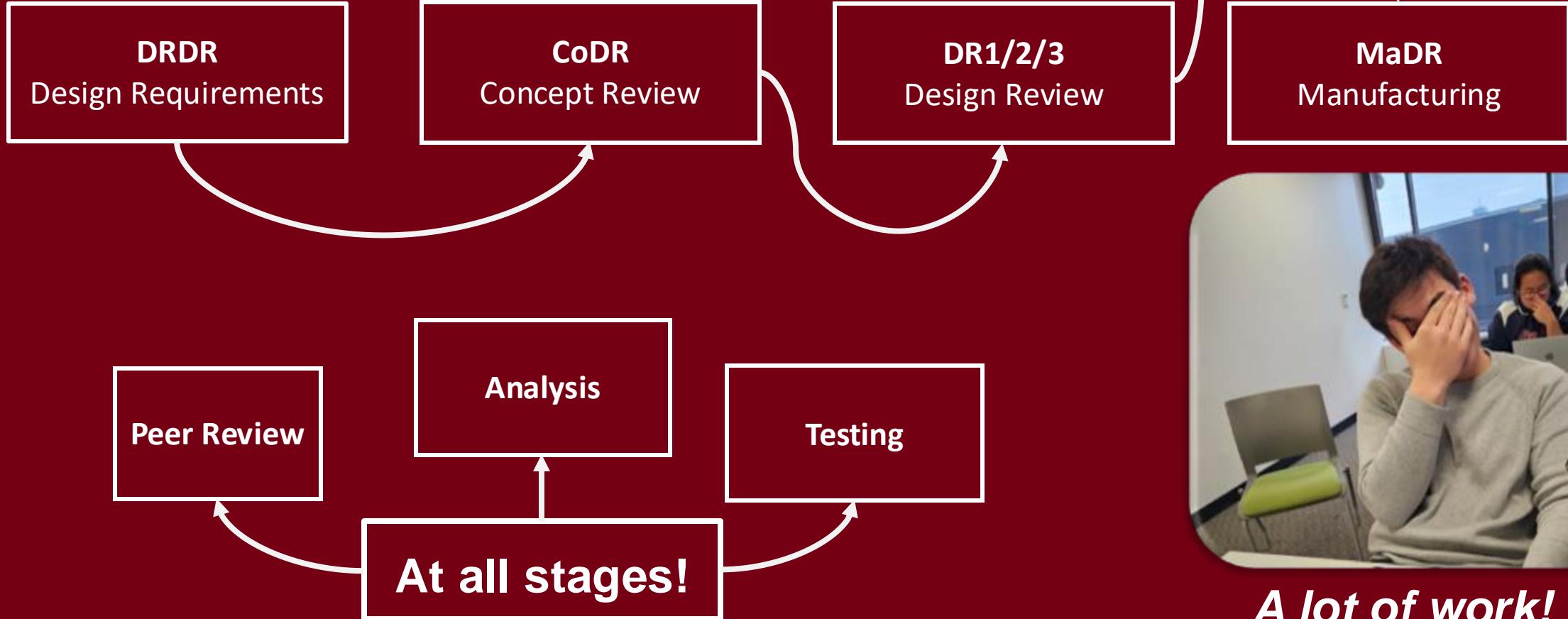


Let's follow a part!

- Upright breakdown. How did we get here?
- There's a lot of complexity – let's go through it piece by piece



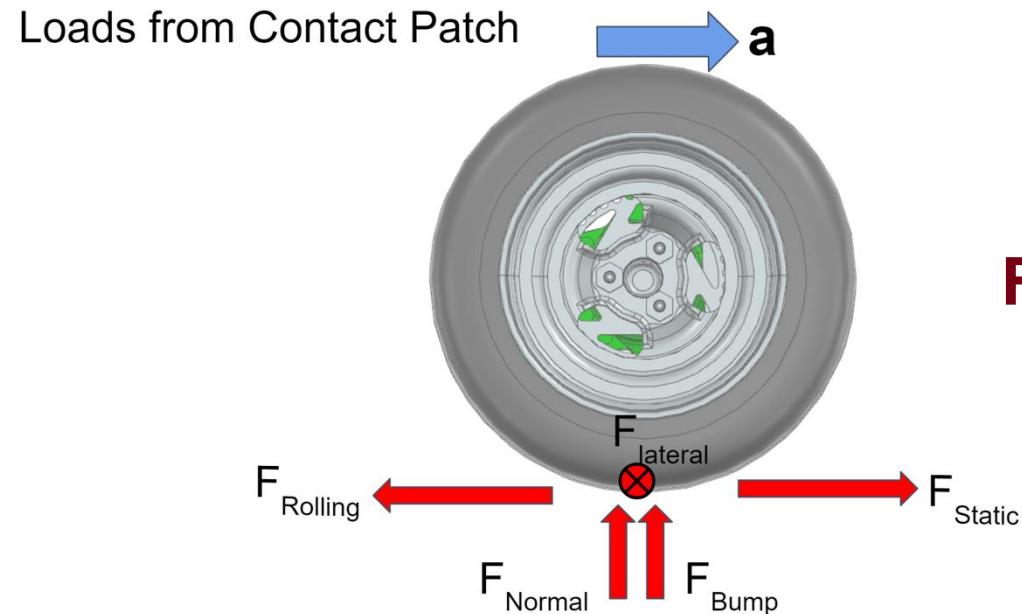
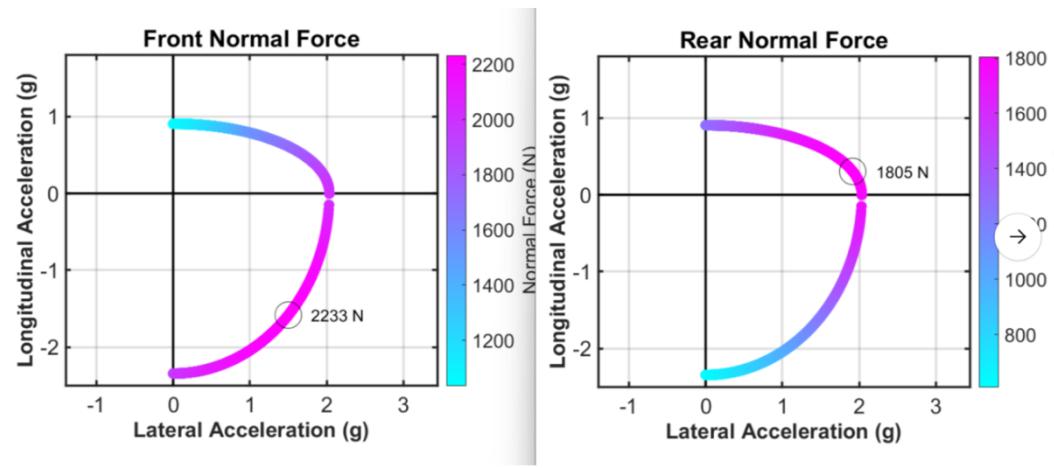
Design Process - Mirrors industry



A lot of work!

DRDR

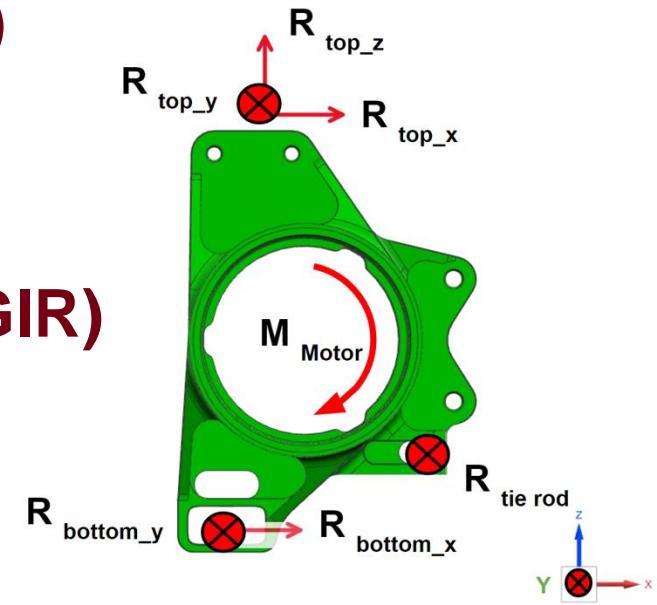
- Design Requirements
- Quantify what you don't know



Initial Simulation

(F vs g's)

FBDs (8.01 GIR)

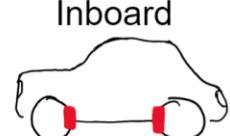
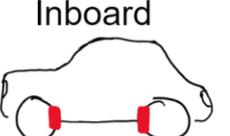
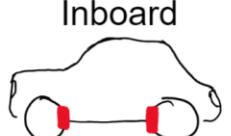


CoDR

- Concept Review
- Evaluate risks and tradeoffs

Risk Matrix For 3D Printing + Billet:

Risk	Likelihood (1-5)	Severity (1-5)	Risk Magnitude	Mitigation
Weaker/Directional Material used can break	2	5	10	Need to pay attention on orientation of print and sizing of part. (1.5-2 SF)
Post machining process messed up	2	5	10	Need to ensure that it is done properly out of house/insured?
Joint calculations are complex and get messed up/break	4	4	16	Need smart people to make sure that all calcs and FEA are done properly (scrutinized)
Bolt Tear Out of 3D printed material	3	4	12	Need to do calcs and testing validation on instron to ensure OK

Design Parameter	Concept 1	Concept 2	Concept 3	Concept 4
Camber Adjust	Slotted Shims	Slotted Shims	Slotted Shims	Slotted Shims
Brake Caliper Location	Inboard 	Inboard 	Inboard 	Inboard 
Manufacturability	3D Printed	Billet Machined	3d Printed w/ Billet portions	Tab and Slot
Camber Clevis Location	Top	Top	Top	Top



Get some real-life data!

DR1/2/3

- Hand calculations
- First principles – 2.001. Quick order-of-magnitude estimate

Assuming SAE Grade 8						
Inputs:						
Value Name:	Value:	Units	Value:	Units		
Tensile Strength:	1034	[Mpa]	1034000000	[Pa]		
Shear Strength:	620.4	[Mpa]	620400000	[Pa]		
Factor of Safety:	1.5	n/a				
General Car Forces						
Rear						
Force Name:	Value	Units	Value	Units		
Max Lateral(+/-)	3.4	[kN]	3400	[N]		
Max Longitudinal	2	[kN]	2000	[N]		
??	-1.8	[kN]	-1800	[N]		
Max Normal	2.3	[kN]	2300	[N]		
Max Bump	3	[kN]	3000	[N]		
Input Forces						
Symbol	Value	Units	Descr.	Source		
Fr_T_Brake	425	Nm	Front Brake Moment (M_y+)	Sam		
Re_T_Brake	205	Nm	Rear Brake Moment (M_y+)	Sam		
T_Tie_Rod	100	Nm	Tie Rod Moment (M_y+)	Suspension		
T_UBJ	x	Nm	Upper Ball Joint Moment (M_y+)	Suspension		
T_LBJ	x	Nm	Lower Ball Joint Moment (M_y+)	Suspension		
Derived Forces						
Symbol	Value	Units	Descr.	Source		
FF_Caliper	1257.83	N	Force at the Front Brake Caliper	Derived		
FR_Caliper	2578.55	N	Force at the Rear Brake Caliper	Derived		
F_Tie_Rod	1238.44	N	Force at Tie Rod	Derived		
F_UBJ	x	N	Force at Upper Ball Joint	Derived		
F_LBJ	x	N	Force at Lower Ball Joint	Derived		
7075-T6 Material Inputs						
Symbol	Value	Units	Converted Val.	Units2	Description	Source
T_Yield	58	ksi	399910000	Pa	Tension Yield	MMPDS
S_Yield	36	ksi	248220000	Pa	Shear Yield	MMPDS
Br_Yield (e)	114	ksi	786030000	Pa	Bearing Yield	MMPDS
FoS	1.5	n/a	1.50	n/a	Factor of Safety	Powertrain
Dimensions						
Symbol	Value	Units	Value	Units	Description	Source
Bolt_CR	0.13	in	0.0795	m	Caliper Bolt Circle Radius	CAD
Bolt_OD	0.3125	in	0.00794	m	Caliper Bolt Outer Diameter	CAD
C_fab_t	0.2	in	0.00508	m	Caliper Tab Thickness	CAD
Tab_OD	0.625	in	0.0159	m	Caliper Tab Outer Diameter	CAD
TR_Dist	3.179	in	0.08075	m	Tie Rod Distance	CAD
TR_Bolt_O	0.1875	in	0.004763	m	Tie Rod Bolt Outer Diameter	CAD
TR_Tab_t	0.1	in	0.00254	m	Tie Rod Tab Thickness	CAD
TR_Tab_O	0.375	in	0.009525	m	Tie Rod Outer Diameter	CAD

Margins			
Margins	Failure Mode	Component	Feature
3.632072625	Tearout	Front Upright	Toe Tie Rod
8.264145249	Tension	Front Upright	Toe Tie Rod
6.645987924	Bearing	Front Upright	Toe Tie Rod
4.187221939	Tearout	Rear Upright	Toe Tie Rod
9.374443878	Tension	Rear Upright	Toe Tie Rod
7.562352001	Bearing	Rear Upright	Toe Tie Rod
2.22411535	Tearout	Rear Upright	UBJ
4.504587183	Tension	Rear Upright	UBJ
17.87287034	Bearing	Rear Upright	UBJ
1.660969869	Tearout	Rear Upright	Camber bracket upright attachment
3.543119289	Tension	Rear Upright	Camber bracket upright attachment
11.82977726	Bearing	Rear Upright	Camber bracket upright attachment

- Continue refining concept feasibility
- Get rough sizes of parts – “sizing”

Table 2.7.9.0(b). Design Mechanical and Physical Properties of 7075 Aluminum Alloy Plate												
Thickness, in.	0.250-0.400			0.500-1.000			1.000-2.000			2.000+		
	A	B	C	A	B	C	A	B	C	A	B	C
<i>Mechanical Properties*</i>												
F _u -ksi	77	80	77	77	79	79	77	77	77	76	77	76
L ₁	77	80	77	77	79	79	77	77	77	76	77	76
F _y -ksi	69	70	68	69	70	70	69	69	69	68	69	68
S _y -ksi	67	69	67	67	69	67	66	66	66	65	67	65
S _u -ksi	71	73	70	71	73	70	70	70	70	69	71	69
F _c -ksi	67	71	68	67	70	68	66	66	66	65	67	65
F _b -ksi	71	73	70	71	73	70	70	70	70	69	71	69
F _t -ksi	43	44	44	43	45	43	42	42	42	41	43	41
F _p -ksi (L & L _T)	117	120	117	117	120	117	117	117	117	116	119	116
L ₁ , L ₂ (in D=1.00)	145	150	145	145	150	145	145	145	145	144	148	144
L ₁ , L ₂ (in D=2.00)	114	116	117	114	116	117	114	114	114	113	115	113
L ₁ , L ₂ (in D=3.00)	97	100	100	97	100	100	97	97	97	96	98	96
c, p, r, t, w, x, y, z	9	11	7	9	11	7	9	9	9	8	10	8
E, 10 ⁶ ksi	10.5	10.5	10.5	10.5	10.5	10.5	10.5	10.5	10.5	10.5	10.5	10.5
E, 10 ⁶ ksi	10.5	10.5	10.5	10.5	10.5	10.5	10.5	10.5	10.5	10.5	10.5	10.5
E, 10 ⁶ ksi	10.5	10.5	10.5	10.5	10.5	10.5	10.5	10.5	10.5	10.5	10.5	10.5
Physical Properties	—											
c, p, r, t, w, x, y, z	9	11	7	9	11	7	9	9	9	8	10	8

FEM – Finite Element Modelling

Reality is complicated... down to subatomic interactions

Everything is a spring! Simpler = better and less likely to go wrong

$$V(x) = \begin{cases} 0, & x < 0, \\ V_0, & x \geq 0. \end{cases}$$

$E = \frac{\hbar^2}{2m}$

$$\Psi_1(x) = \frac{1}{\sqrt{k_1}} (A_{+} e^{ik_1 x} + A_{-} e^{-ik_1 x}) \quad x < 0$$

$$\frac{d}{dt} A(t) = \frac{i}{\hbar} [H, A(t)] + \frac{\partial A}{\partial t}$$

$$\Psi_2(x) = \frac{1}{\sqrt{k_2}} (B_{+} e^{ik_2 x} + B_{-} e^{-ik_2 x}) \quad x > 0$$

$$T|j,m\rangle = |T(j,m)\rangle = (-1)^{j-m} |j,m\rangle$$

$$\frac{\partial}{\partial t} \Psi(r,t) = \hat{H} \Psi(r,t)$$

$$|\Psi\rangle AB = \sum_{i,j} c_{ij} |i\rangle A \otimes |j\rangle B$$

$$\int_{-\infty}^{\infty} W(x,p) dp dx$$

$$H_n(x) = (-1)^n e^{x^2} \frac{d}{dx^n} (e^{-x^2})$$

$$\frac{\hbar^2}{m} \frac{d^2 \Psi}{dx^2} = E \Psi$$

$$\Psi(x) = A e^{ikx} + B e^{-ikx}$$

$$U(t) = \exp\left(\frac{-iHt}{\hbar}\right)$$

$$i\hbar \frac{d}{dt} |\Psi(t)\rangle = H|\Psi(t)\rangle$$

$$A(x) = \exp\left(\frac{i}{\hbar} \int X(t) dt\right)$$

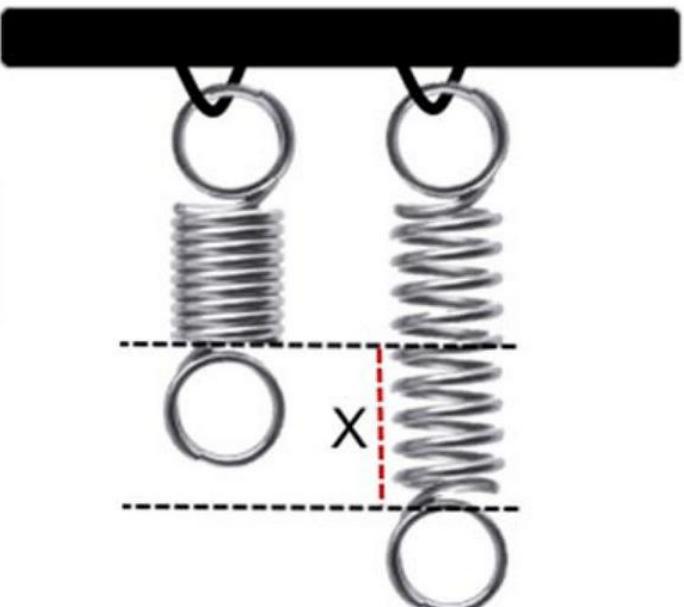
$$P(a,b) = \int d\lambda \cdot \rho(\lambda) \cdot p_a(a,\lambda) \cdot p_b(b,\lambda)$$

$$W \rightarrow \frac{1}{(\pi\hbar)^3} \exp\left[-\alpha^2 \left(x - \frac{pt}{m}\right)^2\right]$$

$$E > V \quad E < V \quad E = V$$

VS

$F = kX$



FEM – Finite Element Modelling



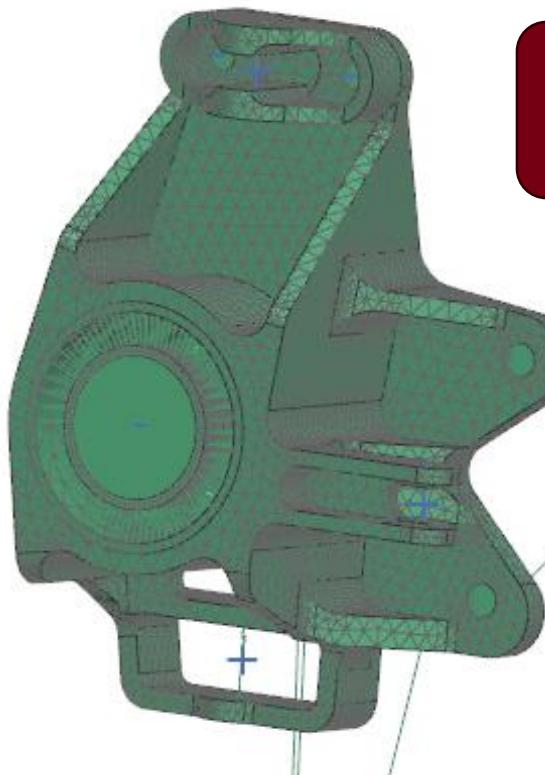
What if we used *millions of springs*?



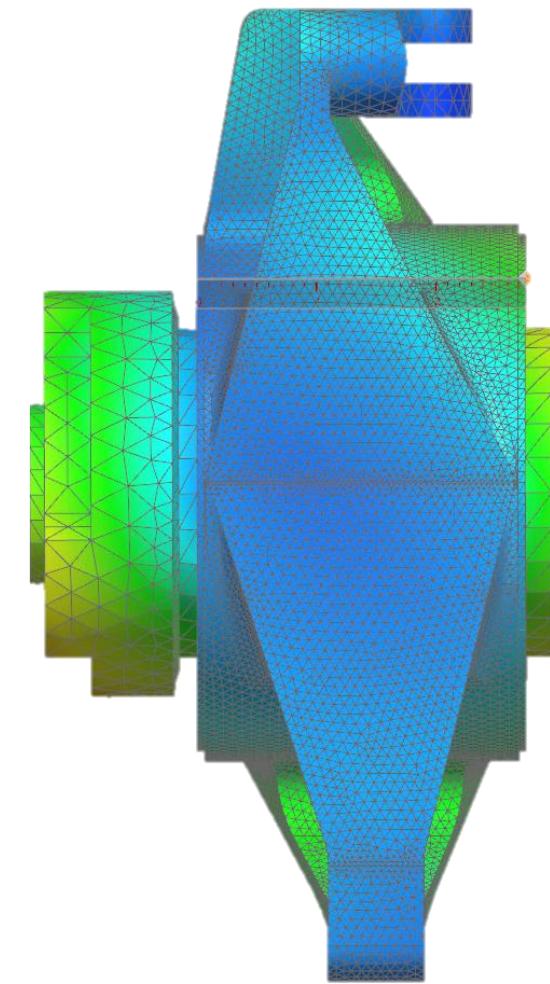
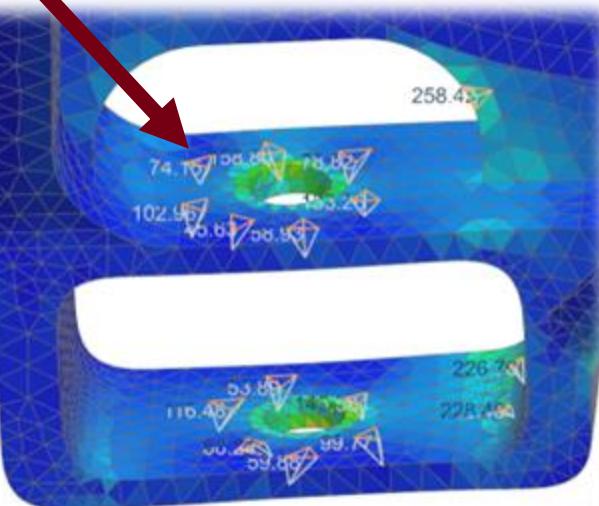
Computers can do math – fast!



“All models are wrong, some are useful” – George Box



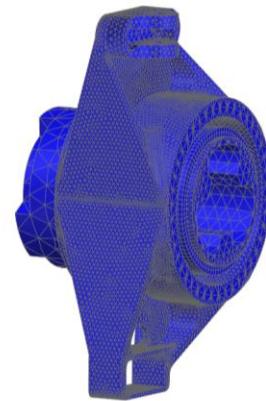
Tons of tiny springs



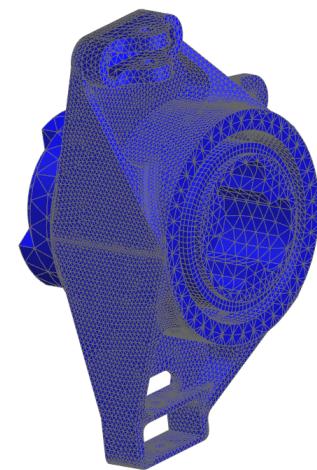
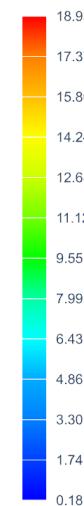
Simulated
Displacement w/BCs

Simulation – Modal Analysis

Animation Frame 1 of 8

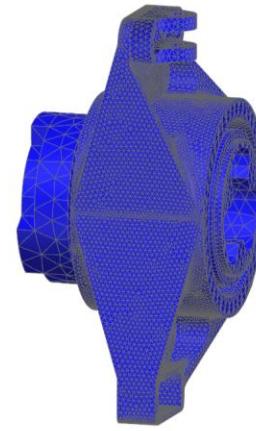
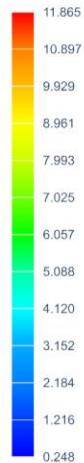


250Hz

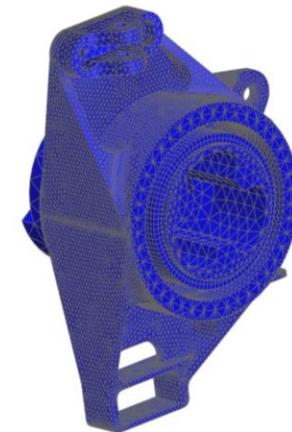


524Hz

Animation Frame 1 of 8



636Hz



905Hz

Resonant failure modes

Treat upright like a tuning fork – what frequencies would it vibrate at?

First-order specific
(mass-weighted) stiffness

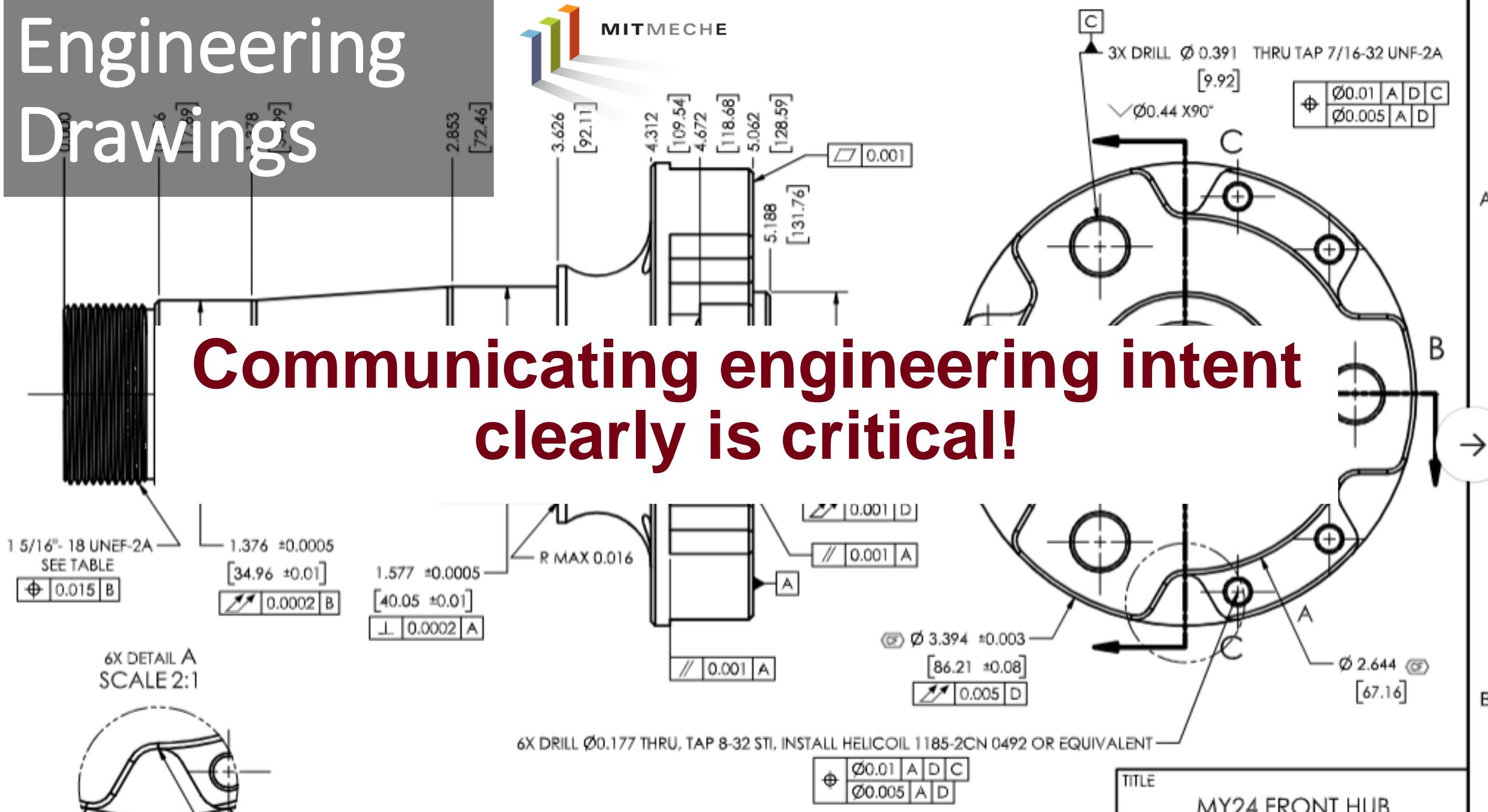
Fundamental Frequency

$$F_0 = \frac{1}{2\pi} \sqrt{\frac{k}{m}}$$

Stiffness Mass

Engineering Drawings

Communicating engineering intent clearly is critical!

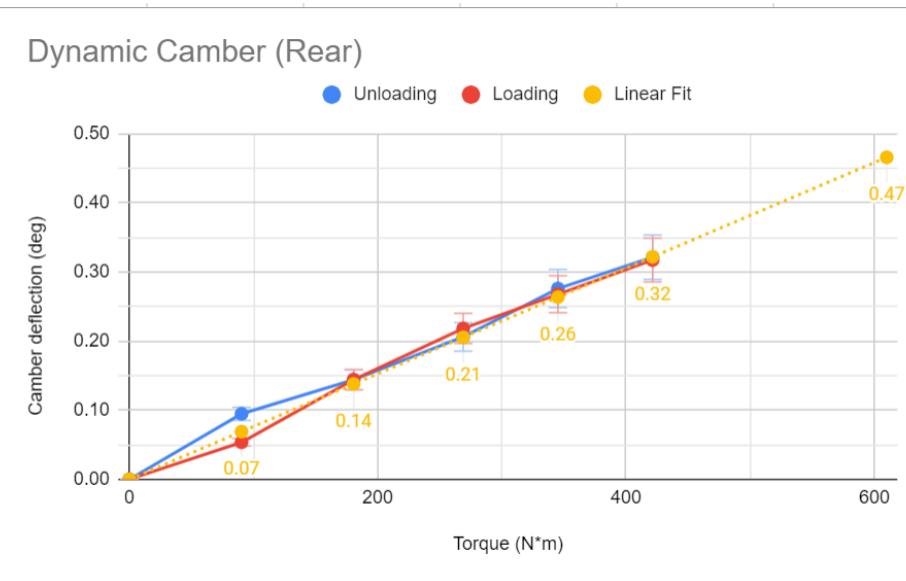
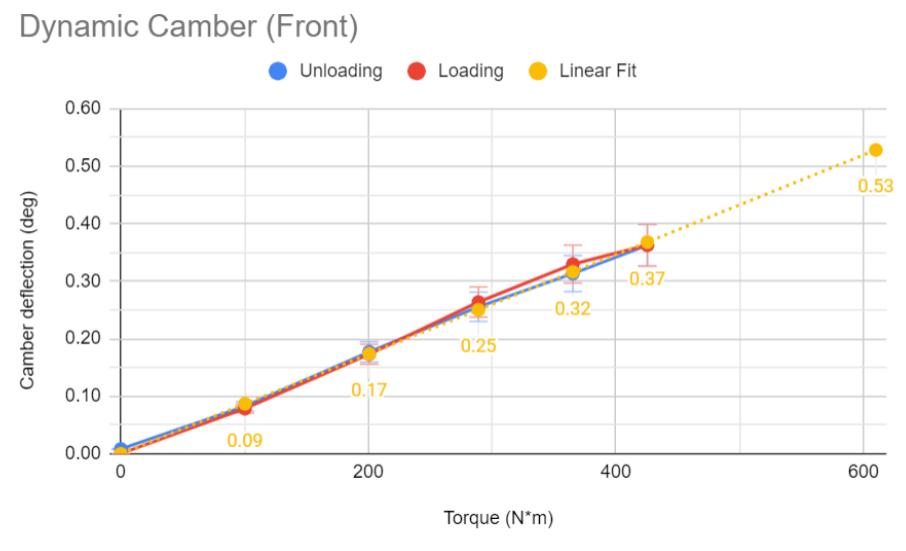


Manufacturing

- Many parts made by students!
 - In N51 – one of many makerspaces on campus!
-



Real World Validation



“Nature cannot be fooled”
- Richard Feynman (MIT B.Sc.)

Shakedown



Put the car through its paces!

**Build capability slowly,
take caution, and check
all your safety systems
first**

*(Testing procedure
needed w/measurable
variables – 2.671)*

Shakedown... oops



Things will break! – “Integration Hell”



Collect & Use Data!

**System as a
whole needs to
be tuned.**

**Full vehicle
dynamics model**

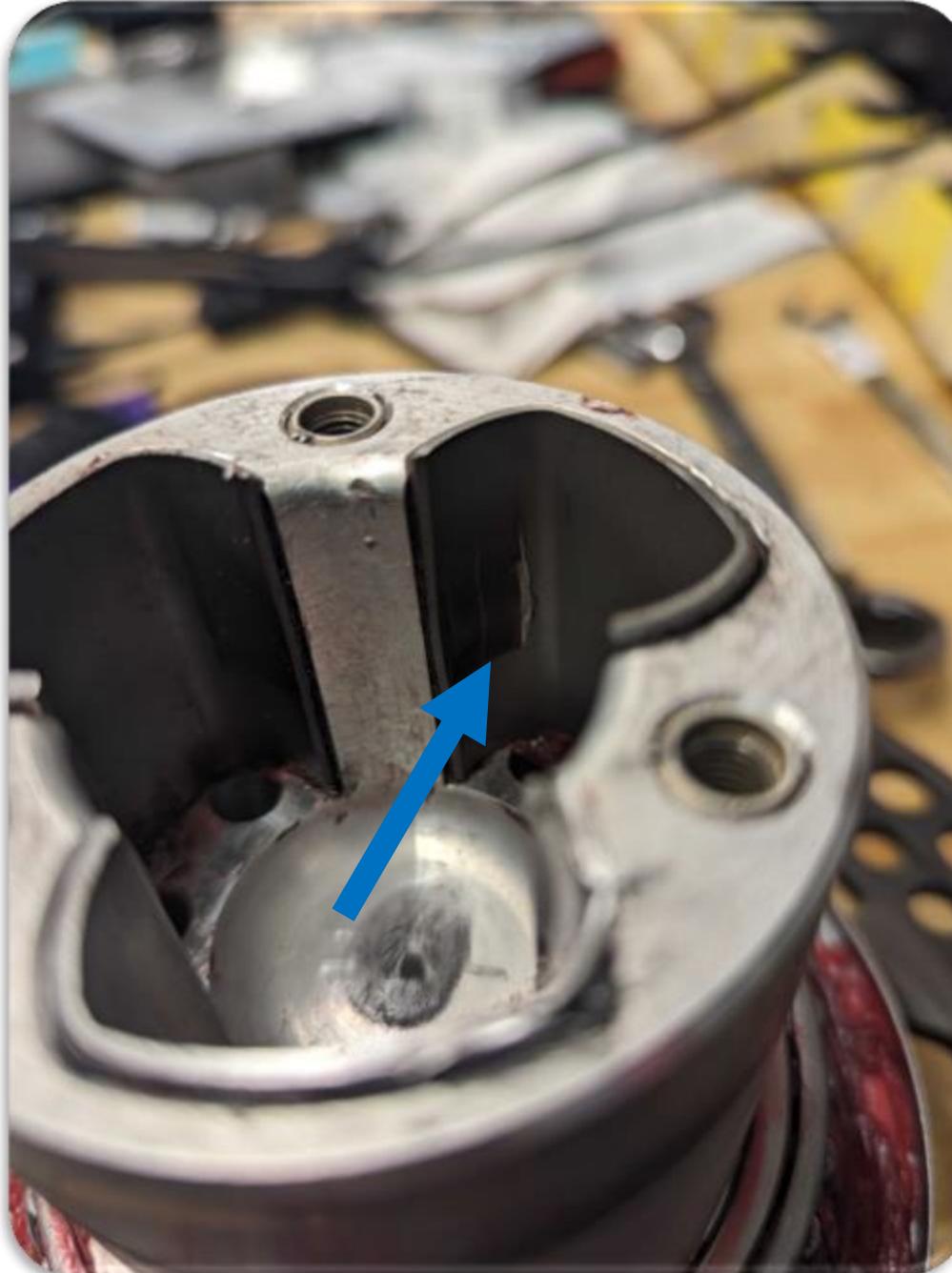
**This is why
teams win** 🏆



Spot it?

No matter how
much validation
you do...

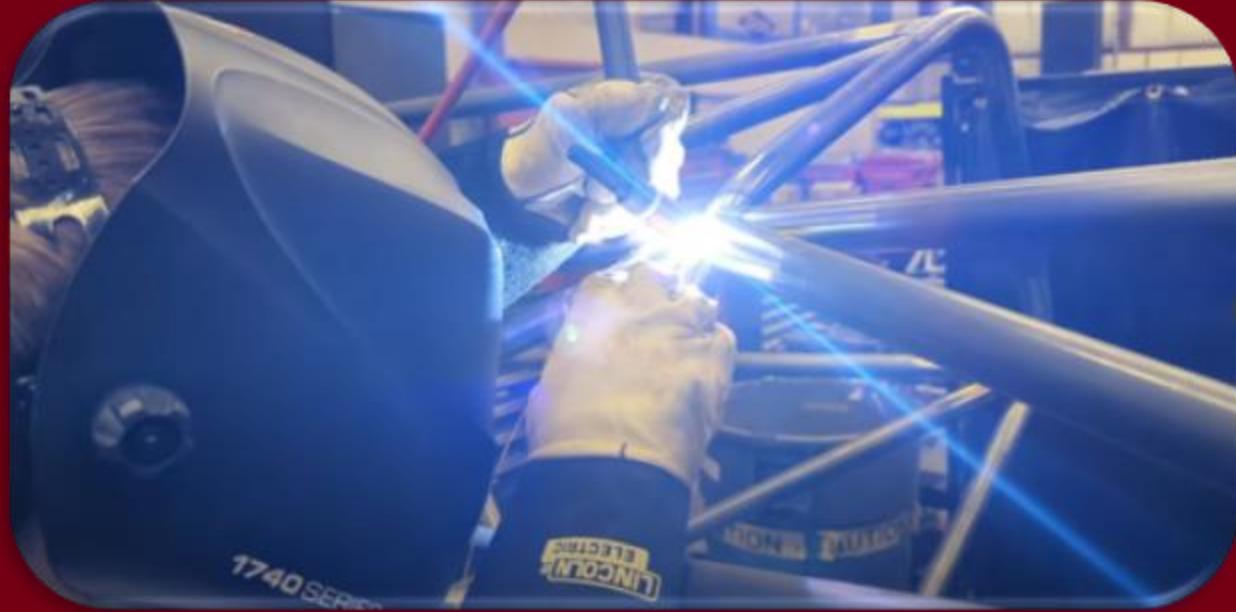
Stress exceeds allowable
due to Hertzian contact
stress, fatigue – 300km



System Highlight - Frame

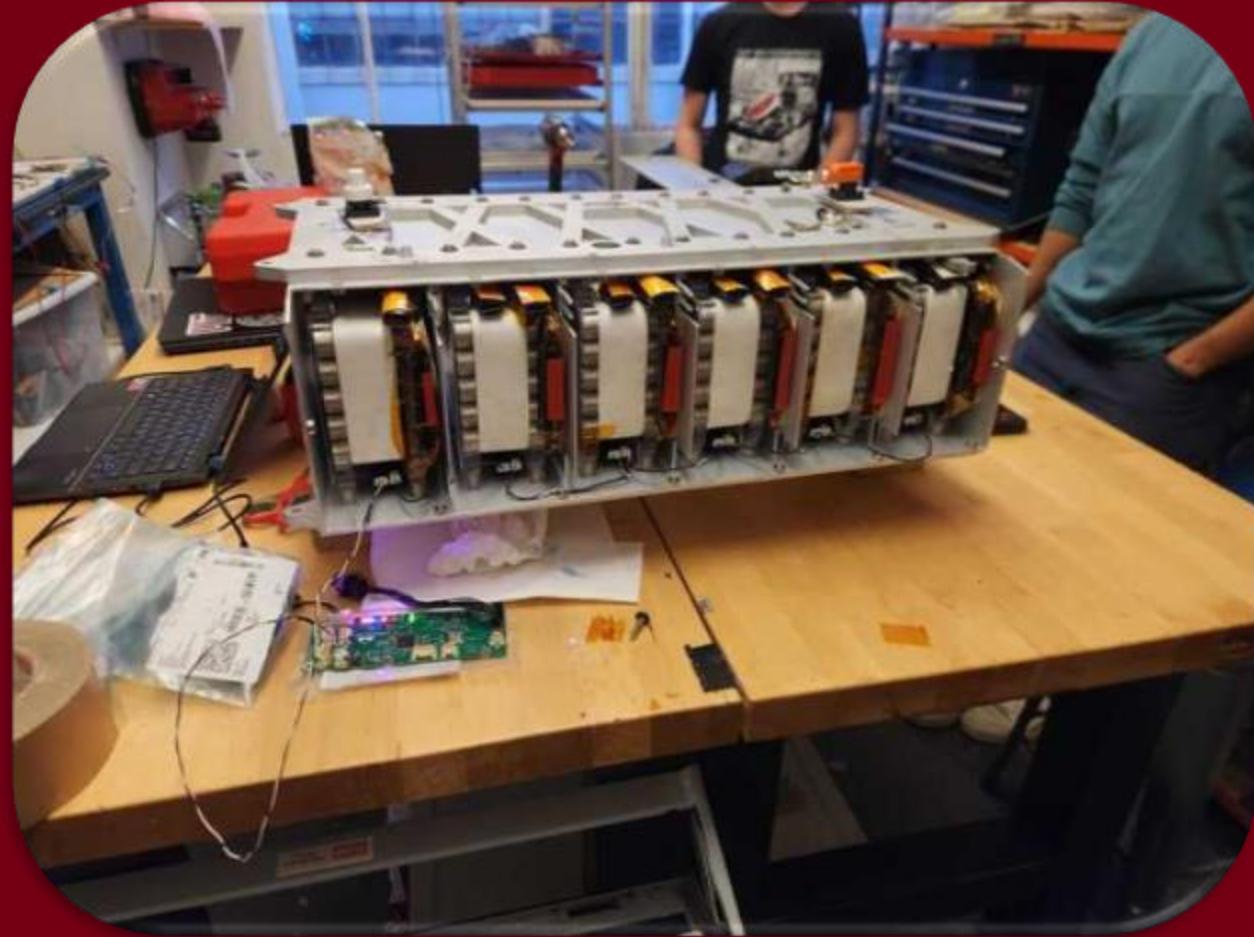


Welded fully in-house in ~35 days (over IAP)



System Highlight - Battery

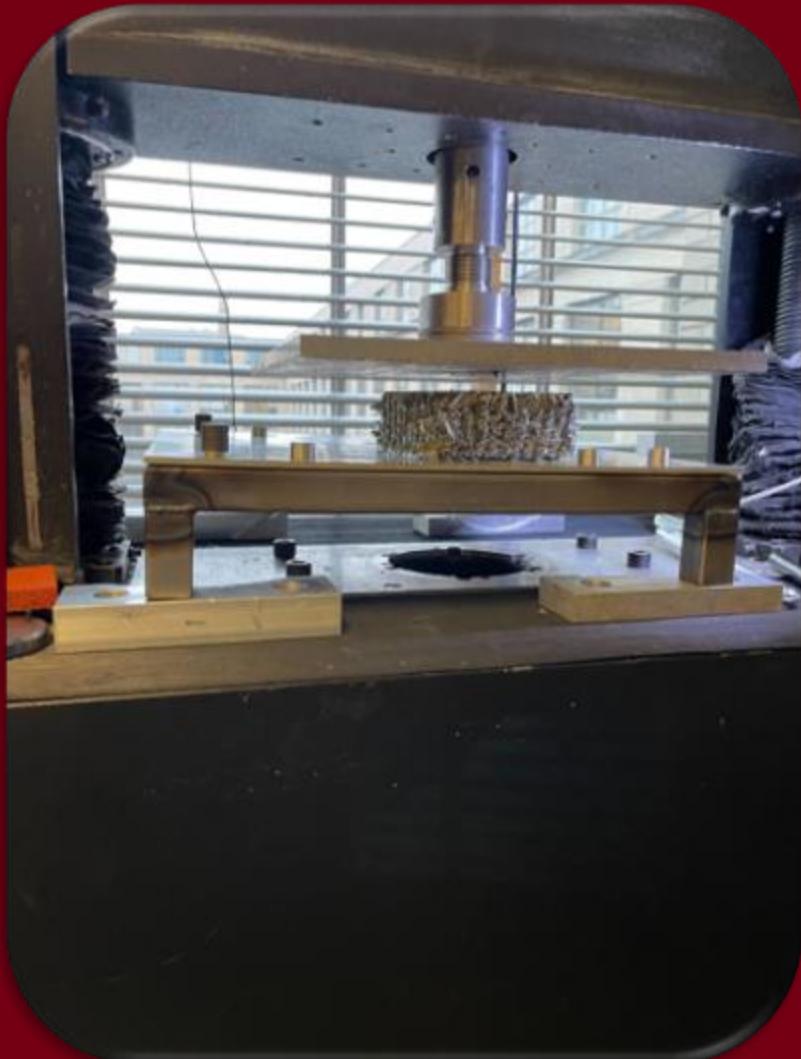
Liquid-cooled, student
designed and built



7.6kWh, 400V, 96s5p, P45B cells

System Highlight – Attenuator

Absorbs impact in event of a
head-on collision. Similar to
Falcon 9 landing legs



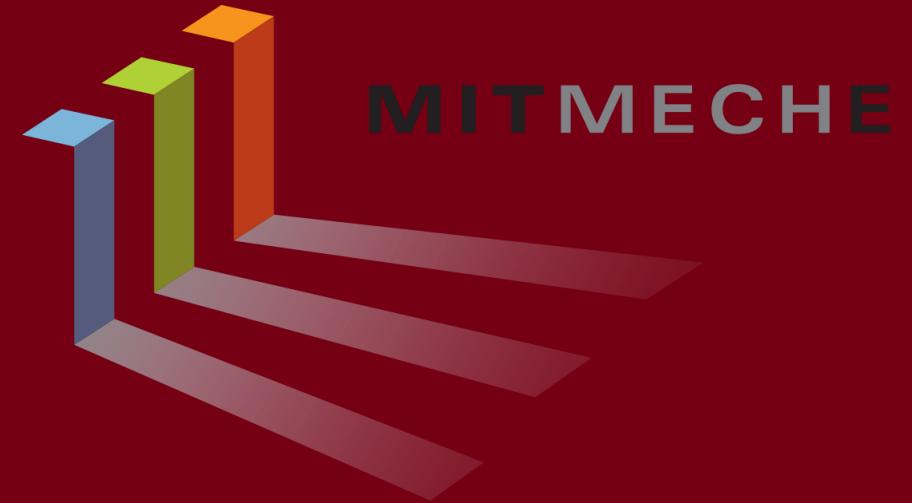
Thanks!!

Henry, Megan, Rakib, Faris, Abe, Sehar, Monica (C/O '25)

Patrick McAtamney (N51 Shop Manager)

Peggy Eysenbach (Shared cookie recipe!)

All Edgerton Center staff!!



“The Senior Hug”

Thanks!!



All my
teammates ☺
Post-Roman

(at 4am)

It's been an
honor



Thanks!!



Accel

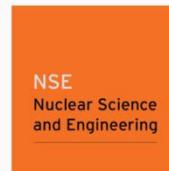


SIEMENS



HENRY FORD

THE CHAN
FAMILY



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BLUE ORIGIN



ALEX SOO



MOZA
RACING

CAMMUS



RALPH & LAURIE
INGLESE