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Systems: Analysis | Linear Systems





Homogeneity

If we scale (increase/decrease the strength of) the input u(t), it is expected that the output function y(t) is also scaled by the same amount.

Mathematically, for some $\alpha \in \mathbb{R}$, $\alpha u(t) \longrightarrow \alpha y(t)$.

Additivity

If $u_1(t) \longrightarrow y_1(t)$, $u_2(t) \longrightarrow y_2(t)$, ..., $u_n(t) \longrightarrow y_n(t)$, as well as $\sum_{k=0}^n u_k(t) \longrightarrow \sum_{k=0}^n y_k(t)$, that is, the output corresponding to the sum of various inputs, is the sum of the individual outputs for individual inputs.

Systems: Analysis | Linear Systems



 ${\sf Homogeneity} + {\sf Additivity} \longrightarrow {\sf Superposition}. \ {\sf Superposition} \Rightarrow {\sf Linearity}$

Example

y(t) = tu(t) is linear.

Consider two inputs $u_1(t)$ and $u_2(t)$. The corresponding outputs are $y_1(t)$ and $y_2(t)$. For some arbitrary constants α, β ,

$$\alpha u_1(t) + \beta u_2(t) = \alpha y_1(t)/t + \beta y_2(t)/t$$

$$\Rightarrow \alpha y_1(t) + \beta y_2(t) = t(\alpha u_1(t) + \beta u_2(t))$$

$$\Rightarrow \tilde{y}(t) = t\tilde{u}(t),$$

which is the same form as y(t) = tu(t). In fact $\tilde{y}(t)$ represents the linear combination of y(t) = tu(t).

Systems: Analysis | Laplace Transform



Example

Consider the mass-spring-damper system described by the differential equation

$$M\frac{d^2y}{dt^2} + b\frac{dy}{dt} + ky = u(t), \ y(0) = 0, \ \dot{y}(0) = 0$$

We wish to obtain the response of the system, y(t), subjected to input force u(t).

Take the Laplace transform of both sides, resulting in

$$Ms^2Y(s) + bsY(s) + kY(s) = U(s).$$

On simplification, we have

$$\frac{Y(s)}{U(s)} = \frac{1}{Ms^2 + bs + k}.$$

- Transfer function is a frequency domain approach.
- Transfer function approach can't be used when there is nonlinearity.

Modern Control: Concept of state



- State of the system: something that characterizes the *past*, *present*, and *future* of the dynamical system.
- A minimal set of variables, such that the knowledge of these variables at any time t_0 , combined with the information on the input applied, are sufficient to determine the behavior of the system at time $t > t_0$.

Example

Consider an on-off switch.

- The switch can be either on or off.
- The state of the switch can assume one of the two possible values.
- If we know the present *state* (position) of the switch at t_0 , we can determine the *state* (position) of the switch at $t > t_0$ if applied input is known.

Modern Control: Concept of state



Example

Classic example: A mass-spring-damper system.

$$M\frac{d^2y}{dt^2} + b\frac{dy}{dt} + ky = u(t).$$

- Position and velocity are sufficient to describe this system.
- Hence, the position and velocity are regarded as state variables for this system.
- Position $\rightarrow x_1$ and velocity $\rightarrow x_2$.

$$\dot{x}_1 = x_2$$

 $\dot{x}_2 = -(b/M)x_2 - (k/M)x_1 + (1/M)u.$

Modern Control: Concept of state



- State variables are not necessarily physical variables.
- Quantities that are not physically measurable or observable can also be chosen as state variables.
- Choice of state variables, and hence, the state model description is not unique.
- The number of states needed to describe a system completely is unique.
- Storage/memory elements in a system can be taken as state variables.
- The output of integrators in a continuous-time system can be taken as state variables.

Modern Control: State Space Description



A system described by nonlinear differential equations

$$\dot{x}_1 = f_1(x_1, x_2, \dots, x_n; u_1, u_2, \dots, u_m)
\dot{x}_2 = f_2(x_1, x_2, \dots, x_n; u_1, u_2, \dots, u_m)
\vdots
\dot{x}_n = f_n(x_1, x_2, \dots, x_n; u_1, u_2, \dots, u_m)$$

i.e.,
$$\dot{\mathbf{x}} = \mathbf{f}(\mathbf{x}; \mathbf{u})$$
, where $\dot{\mathbf{x}} = [\dot{x}_1 \ \dot{x}_2 \ \dots \ \dot{x}_n]^{\mathrm{T}}$ and $\mathbf{f}(\mathbf{x}) = [f_1(\cdot) \ f_2(\cdot) \ \dots \ f_n(\cdot)]^{\mathrm{T}}$. $\mathbf{x} \in \mathbb{R}^n$, $\mathbf{u} \in \mathbb{R}^m$, and $m \le n$.

Modern Control: State Space Description



For a very small deviation, we can expand each equation in Taylor series as

$$\frac{d}{dt}(x_i + \delta x_i) = f_i(x + \delta x; u + \delta u)$$

$$\approx f_i(x; u) + \frac{\partial f_i}{\partial x} \delta x + \frac{\partial f_i}{\partial u} \delta u$$

where

$$\frac{\partial f_i}{\partial x} = \left[\frac{\partial f_i}{\partial x_1} \frac{\partial f_i}{\partial x_2} \dots \frac{\partial f_i}{\partial x_n} \right].$$

As $\frac{d}{dt}(x_i) = f_i(x, u)$, we have

$$\frac{d}{dt}(\delta x_i) \approx \frac{\partial f_i}{\partial x} \delta x + \frac{\partial f_i}{\partial u} \delta u.$$

Modern Control: State Space Description



Combining all n equations,

$$\frac{d}{dt}(\delta x) = \begin{bmatrix} \frac{\partial f_1}{\partial x} \\ \frac{\partial f_2}{\partial x} \\ \vdots \\ \frac{\partial f_n}{\partial x} \end{bmatrix} \delta x + \begin{bmatrix} \frac{\partial f_1}{\partial u} \\ \frac{\partial f_2}{\partial u} \\ \vdots \\ \frac{\partial f_n}{\partial u} \end{bmatrix} \delta u = A(t)\delta x + B(t)\delta u,$$

where

$$\mathsf{A}(t) = \begin{bmatrix} \frac{\partial f_1}{\partial x_1} & \frac{\partial f_1}{\partial x_2} & \dots & \frac{\partial f_1}{\partial x_n} \\ \frac{\partial f_2}{\partial x_1} & \frac{\partial f_2}{\partial x_2} & \dots & \frac{\partial f_2}{\partial x_n} \\ \vdots & & & \vdots \\ \frac{\partial f_n}{\partial x_1} & \frac{\partial f_n}{\partial u_2} & \dots & \frac{\partial f_n}{\partial u_m} \end{bmatrix} \quad \text{and} \quad \mathsf{B}(t) = \begin{bmatrix} \frac{\partial f_1}{\partial u_1} & \frac{\partial f_1}{\partial u_2} & \dots & \frac{\partial f_1}{\partial u_m} \\ \frac{\partial f_2}{\partial u_1} & \frac{\partial f_2}{\partial u_2} & \dots & \frac{\partial f_2}{\partial u_m} \\ \vdots & & & \vdots \\ \frac{\partial f_n}{\partial u_1} & \frac{\partial f_n}{\partial u_2} & \dots & \frac{\partial f_n}{\partial u_m} \end{bmatrix}.$$

Modern Control: State Space Description



For nonlinear measurement equation, y=g(x;u), we can proceed in a similar way such that

$$\delta y = \begin{bmatrix} \frac{\partial g_1}{\partial x} \\ \frac{\partial g_2}{\partial x} \\ \vdots \\ \frac{\partial g_p}{\partial x} \end{bmatrix} \delta x + \begin{bmatrix} \frac{\partial g_1}{\partial u} \\ \frac{\partial g_2}{\partial u} \\ \vdots \\ \frac{\partial g_p}{\partial u} \end{bmatrix} \delta u = C(t) \delta x + D(t) \delta u,$$

With slight abuse of notation, the state and the output equations can be written in matrix form as

$$\dot{x}(t) = A(t)x(t) + B(t)u(t),$$

$$y(t) = C(t)x(t) + D(t)u(t).$$

Note: If the system is operated around only one set point (LTI system), then the matrices A(t), B(t), C(t), D(t) are constant.

Modern Control: State Space Description



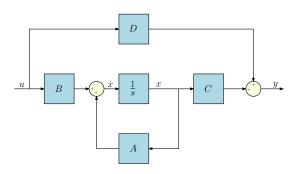


Figure: Illustrative block diagram of LTI state-space model.

 $x \in \mathbb{R}^n$, $u \in \mathbb{R}^m$, $y \in \mathbb{R}^p$.

A: System matrix $(dim[A] = n \times n)$, B: Input matrix $(dim[B] = n \times m)$,

C: Output matrix $(\dim[C] = p \times n)$, D: Feedforward matrix $(\dim[D] = p \times m)$.

Eigenvalues of the system matrix



- Eigenvalues of $A \in \mathbb{R}^{n \times n}$: Roots of the characteristic equation $|\lambda I A| = 0$.
- Eigenvalues are also known as characteristic roots.
- Eigenvalues of the system matrix $A \implies$ the poles of the system.
- For a system to be stable, poles of the system must have negative real parts (poles must lie in left half complex plane).
- Similarly, eigenvalues of the system matrix must have negative real parts for a system to be stable.
- If $A \in \mathbb{R}^{n \times n}$, then a nonzero vector $\mathbf{x} \in \mathbb{R}^n$ is called an eigenvector of A if $A\mathbf{x}$ is a scalar multiple of \mathbf{x} , i.e., $A\mathbf{x} = \lambda \mathbf{x}$ for some scalar λ .
- This scalar λ is called an eigenvalue of A, and ${\bf x}$ is referred as an eigenvector corresponding to λ .
- Any nonzero scalar multiple of an eigenvector is another eigenvector.

Similarity Transformation and diagonalization



- If A has linearly independent eigenvectors, then there exists an invertible matrix T that diagonalizes A.
- \bullet Resulting diagonal matrix: $\tilde{\mathsf{A}} = \mathsf{T}^{-1}\mathsf{A}\mathsf{T}$
- Determinant of A and \tilde{A} are same. How? Proof: $\det(\tilde{A}) = \det(T^{-1}AT) = \det(T^{-1})\det(A)\det(T) = \det(A)$.
- Eigenvalues of A and A are same. How?

$$\begin{split} |\lambda I - T^{-1}AT| &= |\lambda T^{-1}T - T^{-1}AT| \\ &= |T^{-1}(\lambda I - A)T| \\ &= |T^{-1}| |\lambda I - A| |T| \\ &= |T^{-1}| |T| |\lambda I - A| \\ &= |T^{-1}T| |\lambda I - A| \\ &= |\lambda I - A|. \end{split}$$

Similarity Transformation and diagonalization



- Matrices A and \tilde{A} are similar if there exists an invertible matrix T such that $\tilde{A} = T^{-1}AT$.
- T can be formed by stacking the columns of T by linearly independent eigenvectors of A.
- If A has fewer than *n* linearly independent eigenvectors, then it can't be diagonalized.

Example

Let
$$A = \begin{bmatrix} 0 & 0 & -2 \\ 1 & 2 & 1 \\ 1 & 0 & 3 \end{bmatrix}$$
. The characteristic eq. is $(\lambda - 1)(\lambda - 2)^2 = 0$. However,

there are three linearly independent eigenvectors $v_1 = [-1 \ 0 \ 1]^{\rm T}$, $v_2 = [0 \ 1 \ 0]^{\rm T}$

and
$$v_3 = [-2 \ 1 \ 1]^{\mathrm{T}}.$$
 Thus, $T = \begin{bmatrix} -1 & 0 & -2 \\ 0 & 1 & 1 \\ 1 & 0 & 1 \end{bmatrix}$, $T^{-1}AT = \tilde{A} = \begin{bmatrix} 2 & 0 & 0 \\ 0 & 2 & 0 \\ 0 & 0 & 1 \end{bmatrix}.$

Solution of the homogeneous state equation



- Linear homogeneous state equation, $\dot{x} = Ax$.
- Let $\Phi(t)$ be a $n \times n$ matrix that satisfies the linear homogeneous equation,

$$rac{d\Phi(t)}{dt}=\mathsf{A}\Phi(t).$$

- $\Phi(t)$ is known as the state transition matrix, i.e, it describes the evolution of state as time progresses.
- If x(0) denotes the initial state at t = 0, then

$$x(t) = \Phi(t)x(0)$$

is the solution of the linear homogeneous equation for $t \geq 0$.

• How do we compute $\Phi(t)$?

Computation of state transition matrix



• Take the Laplace transform of $\dot{x} = Ax$ to get

$$sX(s) - x(0) = AX(s) \Rightarrow X(s) = (sI - A)^{-1}x(0)$$

 $\Rightarrow x(t) = \mathcal{L}^{-1}[(sI - A)^{-1}]x(0) = \Phi(t)x(0).$

- Scalar differential equation, $\dot{x} = ax$.
- Solution? $x(t) = e^{at}x(0)$.
- Draw analogy to the vector-matrix differential equation, $\dot{x} = Ax$.
- Assume solution of the form $x(t) = e^{At}x(0)$, where

$$e^{At} = \sum_{k=0}^{\infty} \frac{A^k t^k}{k!}.$$

• Verify that e^{At} is indeed a solution to $\dot{x} = Ax$.

$$\frac{d}{dt}e^{At} = Ae^{At} = Ax$$

Computation of state transition matrix



Example

Compute the state transition matrix $(\Phi(t))$ for the given state model.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$$

Here, the system matrix
$$A = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}$$
.

First method:
$$\Phi(t) = \mathcal{L}^{-1}[(sI - A)^{-1}].$$

$$sI - A = \begin{bmatrix} s & 0 \\ 0 & s \end{bmatrix} - \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} = \begin{bmatrix} s & -1 \\ 2 & s+3 \end{bmatrix}$$

$$\Rightarrow (\mathsf{sI} - \mathsf{A})^{-1} = \begin{bmatrix} \frac{\mathsf{s} + \mathsf{3}}{(\mathsf{s} + \mathsf{1})(\mathsf{s} + \mathsf{2})} & \frac{1}{(\mathsf{s} + \mathsf{1})(\mathsf{s} + \mathsf{2})} \\ \frac{\mathsf{2}}{(\mathsf{s} + \mathsf{1})(\mathsf{s} + \mathsf{2})} & \frac{\mathsf{3}}{(\mathsf{s} + \mathsf{1})(\mathsf{s} + \mathsf{2})} \end{bmatrix}$$

$$\Rightarrow (sI - A)^{-1} = \begin{bmatrix} \frac{s+3}{(s+1)(s+2)} & \frac{1}{(s+1)(s+2)} \\ \frac{-2}{(s+1)(s+2)} & \frac{s}{(s+1)(s+2)} \end{bmatrix}$$

$$\therefore \Phi(t) = \mathcal{L}^{-1}[(sI - A)^{-1}] = \begin{bmatrix} 2e^{t} - e^{2t} & e^{t} - e^{2t} \\ -2e^{t} + 2e^{2t} & -e^{t} + 2e^{2t} \end{bmatrix}$$

Computation of state transition matrix



Example

Second method: Power series expansion of matrix exponential.

$$\Phi(t) = \mathsf{e}^{\mathsf{A}t} = \sum_{k=0}^{\infty} rac{\mathsf{A}^k t^k}{k!} = \mathsf{I} + \mathsf{A}t + rac{\mathsf{A}^2 t^2}{2} + rac{\mathsf{A}^3 t^3}{6} + \dots$$

$$\Phi(t) = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} + \begin{bmatrix} 0 & t \\ -2t & -3t \end{bmatrix} + \begin{bmatrix} -t^2 & -3t^2/2 \\ 3t^2 & 7t^2/2 \end{bmatrix} + \cdots$$
$$= \begin{bmatrix} 2e^t - e^{2t} & e^t - e^{2t} \\ -2e^t + 2e^{2t} & -e^t + 2e^{2t} \end{bmatrix}$$

- The power series method may be cumbersome sometimes.
- Can we compute e^{At} by another method?
- Cayley-Hamilton theorem: every square matrix satisfies its own characteristic equation.

$$f(\lambda) = \lambda^n + \alpha_1 \lambda^{n-1} + \alpha_2 \lambda^{n-2} + \dots + \alpha_n = 0$$

$$\Delta(A) = A^n + \alpha_1 A^{n-1} + \alpha_2 A^{n-2} + \dots + \alpha_n = 0.$$



Properties of $\Phi(t)$

- $\Phi(t)$ represents the free response of the system, i.e., response due to initial conditions only.
- **2** $\Phi(t)$ is only dependent on A.
- **1** $\Phi(0) = 1$. How?

Proof:
$$\Phi(t) = e^{At} \Rightarrow \Phi(0) = e^{A0} = I$$
.

1 $\Phi^{-1}(t) = \Phi(-t)$. How?

Proof:
$$\Phi(t) = e^{At} \Rightarrow \Phi^{-1}(t) = e^{-At} = \Phi(-t)$$
.

Proof:
$$\Phi(t_2-t_1)\Phi(t_1-t_0)=e^{A(t_2-t_1)}e^{A(t_1-t_0)}=e^{A(t_2-t_0)}=\Phi(t_2-t_0).$$

Proof:
$$\Phi^k(t) = e^{(At)^k} = e^{kAt} = \Phi(kt)$$
.

State transition equation



- Consider LTI state equation $\dot{x} = Ax(t) + Bu(t)$.
- Taking Laplace transform on both sides,

$$sX(s) - x(0) = AX(s) + BU(s)$$

$$\Rightarrow X(s) = (sI - A)^{-1}x(0) + (sI - A)^{-1}BU(s)$$

$$\Rightarrow x(t) = \mathcal{L}^{-1}[(sI - A)^{-1}]x(0) + \mathcal{L}^{-1}[(sI - A)^{-1}BU(s)]$$

$$\Rightarrow x(t) = \Phi(t)x(0) + \underbrace{\int_{0}^{t} \Phi(t - \tau)Bu(\tau)d\tau}_{\text{convolution integral}}$$

• In general, if $t_0 \neq 0$, then the total response is given by

$$\underbrace{\mathsf{x}(t)}_{\text{total response}} = \underbrace{\Phi(t-t_0)\mathsf{x}(t_0)}_{\text{free response}} + \underbrace{\int_{t_0}^t \Phi(t-\tau)\mathsf{Bu}(\tau)d\tau}_{\text{forced response}}.$$

State-space to transfer function



Relation between state-space model and transfer function

$$X(s) = (sI - A)^{-1}x(0) + (sI - A)^{-1}BU(s)$$

$$X(s) = (sI - A)^{-1}x(0) + (sI - A)^{-1}BU(s)$$

$$\Rightarrow X(s) = (sI - A)^{-1}BU(s).$$

- Output equation is given as y(t) = Cx(t) + Du(t).
- In Laplace domain,

$$Y(s) = CX(s) + DU(s)$$

$$= C(sI - A)^{-1}BU(s) + DU(s)$$

$$= \{C(sI - A)^{-1}B + D\}U(s)$$

$$\Rightarrow \frac{Y(s)}{U(s)} = C(sI - A)^{-1}B + D$$

Eigenvalues as modes of the system



- Suppose A has distinct eigenvalues, $\lambda_1, \lambda_2, \ldots, \lambda_n$, so that it has a set of linearly independent eigenvectors, v_1, v_2, \ldots, v_n .
- If $x(0) = x_0 = v_i$, then

$$x(t) = e^{At}x_0 = e^{At}v_j = \sum_{k=0}^{\infty} \frac{A^k t^k}{k!}v_j = \sum_{k=0}^{\infty} \frac{\lambda_j^k t^k}{k!}v_j = e^{\lambda_j t}v_j$$

- If we start with initial condition along an eigenvector of A, the solution x(t) will stay in the direction of the eigenvector, with length being stretched or shrunk by $e^{\lambda_j t}$.
- If the eigenvectors v_1, v_2, \ldots, v_n are linearly independent,

$$x_0 = \sum_{j=1}^n \xi_j v_j, \quad \xi_j = x_0^T v_j$$

Eigenvalues as modes of the system



• For some $\xi = [\xi_1, \dots, \xi_n]^\top$, and $P = [v_1, v_2 \dots, v_n]^\top$, we have

$$x_0 = \sum_{j=1}^n \xi_j v_j = P\xi$$

Solutions of homogenous system

$$x(t) = e^{At}x_0 = e^{At}\sum_{j=1}^n \xi_j v_j = \sum_{j=1}^n \xi_j e^{At}v_j = \sum_{j=1}^n \xi_j e^{\lambda_j t}v_j$$

• Solution x(t): A (time-varying) linear combination of the eigenvectors of A.

Eigenvalues as modes of the system



- Note that P diagonalizes A, that is, $P^{-1}AP = \Lambda$.
- Since P diagonalizes A, we have

$$e^{At}x_0 = Pe^{\Lambda t}P^{-1}x_0 = Pe^{\Lambda t}\xi$$

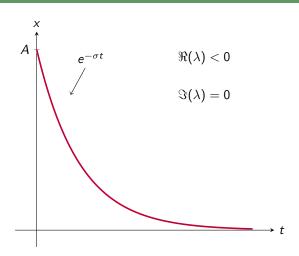
$$= P\begin{bmatrix} \xi_1 e^{\lambda_1 t} \\ \xi_2 e^{\lambda_2 t} \\ \vdots \\ \xi_n e^{\lambda_n t} \end{bmatrix}$$

$$= \sum_j \xi_j e^{\lambda_j t} v_j$$

- Same result as the previous one!
- These representations of the solution x(t) in terms of the eigenvectors are known as the modal decomposition of x(t).

Eigenvalues as modes of the system





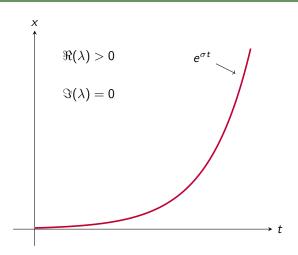
• Purely real eigenvalue, $\lambda = \sigma \pm i \, 0$.

This mode is statically stable.

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Eigenvalues as modes of the system

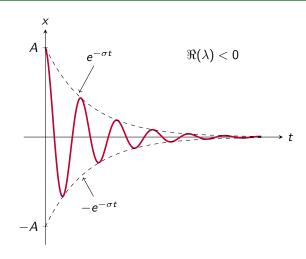




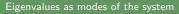
- Purely real eigenvalue, $\lambda = \sigma \pm j$ 0.
- This mode is statically unstable.

Eigenvalues as modes of the system

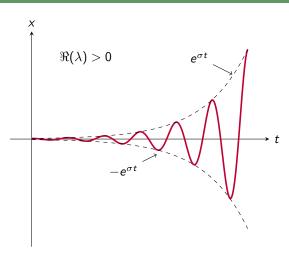




- Complex conjugate eigenvalues, $\lambda = \sigma \pm j\omega$, with negative real part.
- This mode is dynamically stable/damped/convergent.







- Complex conjugate eigenvalues, $\lambda = \sigma \pm j\omega$, with positive real part.
- This mode is dynamically unstable/undamped/divergent.



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Thank you for your attention !!!