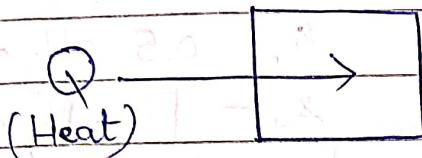


# Heat & Thermometry

## Calorimetry



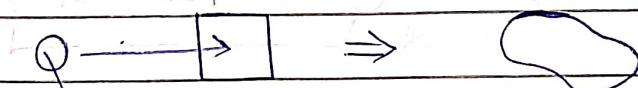
If heat supplied to substance it phase  
NOT change  $\Rightarrow$  Temp. of sub. inc.

$$Q = m s (\Delta T)$$

Heat mass specific heat capacity Temp. inc.  
Temp. change

Specific heat Capacity:

Heat req. to raise temp. of 1 kg of substance by  $1^{\circ}\text{C}$ .

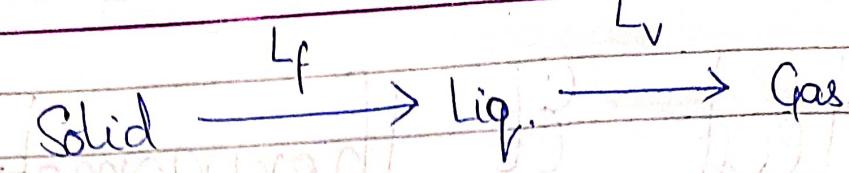


If heat supplied it phase changes  
 $\Rightarrow$  Temp. NOT change

$$Q = m L$$

Heat mass Latent heat

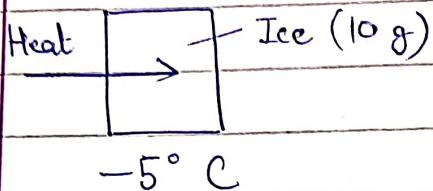
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$L_f$  = Latent heat of fusion ;

$L_v$  = Latent heat of vaporisation

(Q)



$$\delta_{\text{ice}} = 0.5 \text{ cal/g}^\circ\text{C}$$

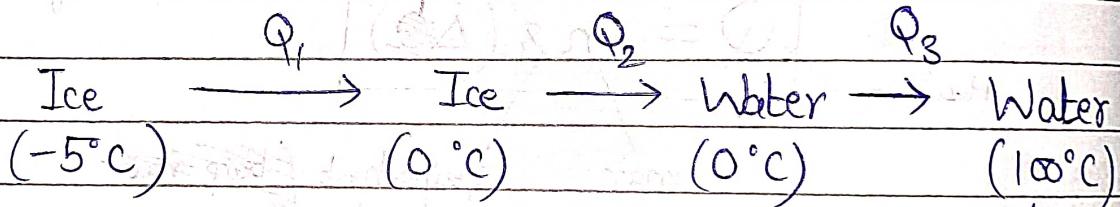
$$\delta_w = 1 \text{ cal/g}^\circ\text{C}$$

$$L_f = 80 \text{ cal/g}$$

$$L_v = 540 \text{ cal/g}$$

Find heat reqd. to convert ice into steam at  $100^\circ\text{C}$ .

A)



$$Q_1 = 10 \cdot 0.5 \cdot 5 = 25 \text{ cal}$$

$$Q_2 = 10 \cdot 80 = 800 \text{ cal}$$

$$Q_3 = 10 \cdot 1 \cdot 100 = 1000 \text{ cal}$$

$$Q_4 = 10 \cdot 540 = 5400 \text{ cal}$$

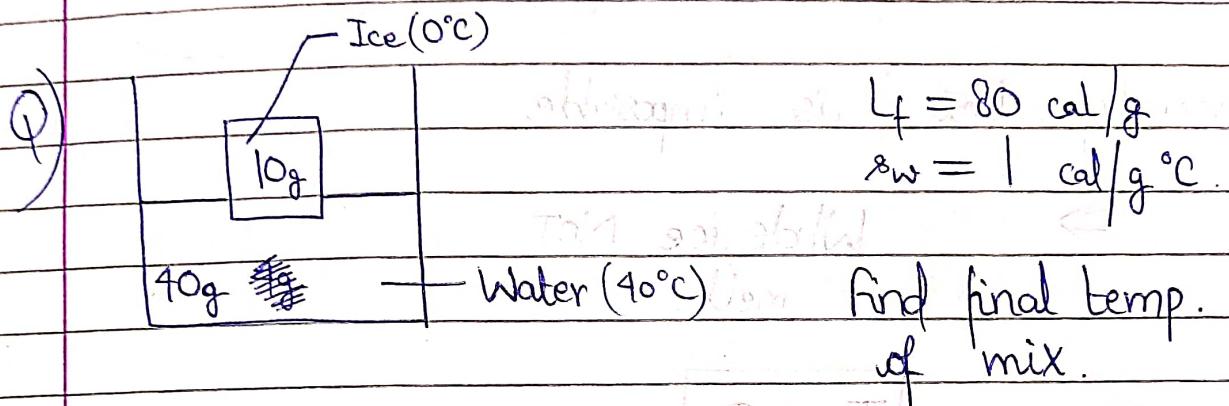
$$Q = \sum Q_i \Rightarrow$$

$$Q = 7225 \text{ cal}$$

# Principle of Calorimetry

$$\text{(Heat Loss)} = \text{(Heat Gain)}$$

by 1 subs.    by other subs.



A) Let final temp. be  $T$

$$Q_{\text{gain(ice)}} = m_{\text{ice}} L_f + m_w \varrho_w \Delta T$$

$$Q_{\text{Loss}} = m_w \varrho_w \Delta T' \quad (\text{water dd})$$

$$\Rightarrow (10)(80) + (10)(1)(T - 0) = (40)(1)(40 - T)$$

$$\Rightarrow 80 + T = 160 - \frac{1}{4}T \Rightarrow T = 16^{\circ}\text{C}$$

Q) In above  $Q$ , if mass of ice is taken  $40\text{ g}$ , find final temp.

A) ★

Since we do NOT know how much ice melts, assume whole melts

$$\Rightarrow (40)(80) + (40)(1)(T-0) = (40)(1)(90-T)$$

$$\Rightarrow 80 + T = (90 - T) \Rightarrow T = (-20)$$

Obviously, this is impossible.

$\Rightarrow$  Whole ice NOT

almost liquid but melt

$$\Rightarrow T_{\text{mix.}} = 0$$

★

If 2 subs. of temp.  $T_1$  &  $T_2$  mix. while calc. — Q

(whole gas not condensee)

$$T > \max \{ T_1, T_2 \} \Rightarrow$$

$$T_{\text{mix.}} = \max \{ T_1, T_2 \}$$

$$(T < \min \{ T_1, T_2 \}) \Rightarrow$$

$$T_{\text{mix.}} = \min \{ T_1, T_2 \}$$

Q)

In above Q, find m of ice melted.

A)

Let m mass melt. Since  $T_{\text{mix.}} = 0$ .

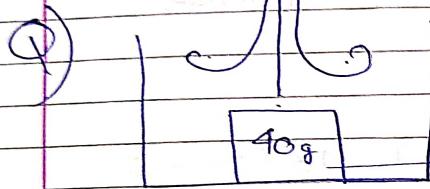
$$\Rightarrow m \cdot L_f = (m_{w, \text{old}}) s_w \Delta T$$

$$\Rightarrow m (80) = (40)(1)(40 - 0)$$

$T_{\text{mix.}}$

$$\Rightarrow m = 20 \text{ g}$$

Steam (10g at  $100^{\circ}\text{C}$ )



$$L_f = 80 \text{ cal/g}$$

$$L_v = 540 \text{ cal/g}$$

$$s_w = 1 \text{ cal/g}^{\circ}\text{C}$$

$\Rightarrow$  find final temp.

A) Assume both whole ice melts & whole steam condenses.

$$\Rightarrow (40)(80) + (40)(1)(T-0) = (10)(540) + (10)(100-T)$$

$$\Rightarrow 320 + 4T = 540 + 100 - T$$

$$\Rightarrow T = 64^{\circ}\text{C}$$

Q) In above Q, if mass of steam is taken 20g, find final temp. Also find mass of steam condensed.

A) Assume whole steam condenses. It whole ice melts.

$$(40)^{(2)}(80) + (40)^{(2)}(1)(T-0) = (20)(540) + (20)(100)$$

$$\Rightarrow 160 + 2T = 540 + 100 - T$$

$$\Rightarrow T = 160 > \max \{100^\circ\text{C}, 0^\circ\text{C}\}$$

$$\Rightarrow T_{\text{mix}} = 100^\circ\text{C}$$

at whole steam does NOT condense.

Let  $m$  mass condense.

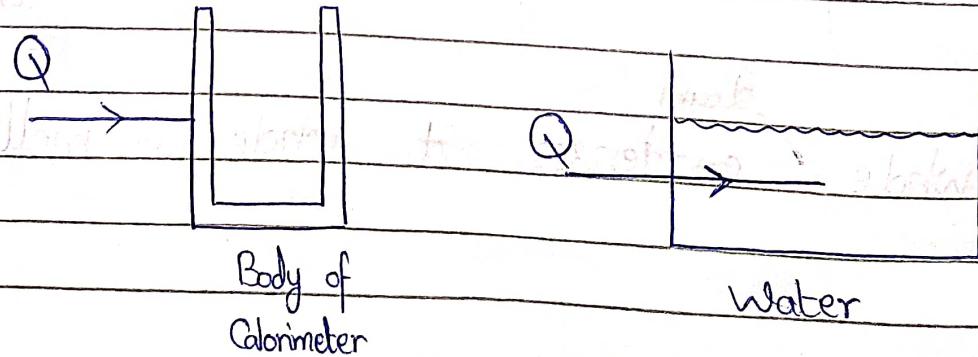
$$\Rightarrow m(540) = (40)(80) + (40)(100 - 0)$$

$$\Rightarrow 54m = 320 + 400$$

$$\Rightarrow m = 40/3 \text{ g}$$

## Water Equivalent

To account for heat loss due to body of calorimeter.



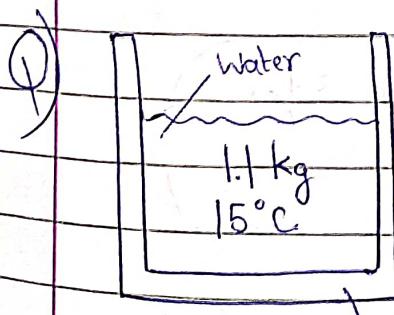
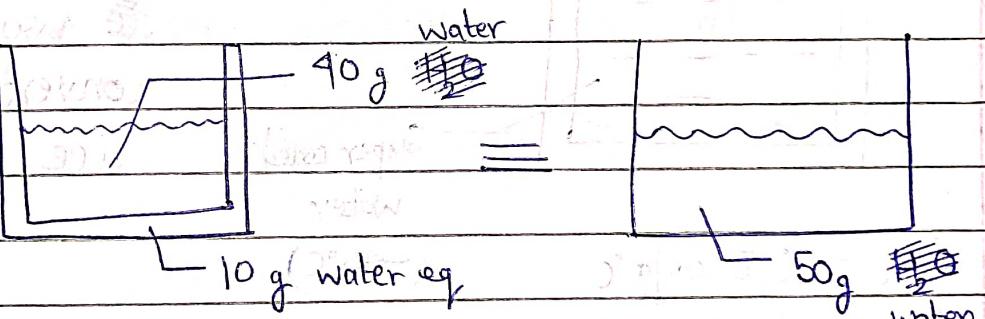
Supply  $Q$  heat to both body of calorimeter & water. We take mass of water in such qty. s.t. rise in temp in both cases is same.

$$\Rightarrow Q = m_c \delta_c \Delta T \quad \text{&} \quad Q = m_w \delta_w \Delta T$$

$$\Rightarrow m_w = m_c \delta_c$$

Water eq.

### Usefulness

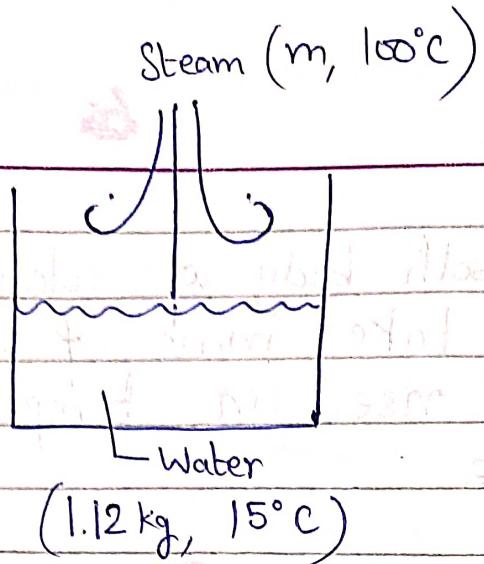


Water is to be raised to  $80^\circ\text{C}$  by passing steam at  $100^\circ\text{C}$ .

find min. amt of steam req.

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A)



$$(m)(540) + (m)(100-80)$$

$$= (1.12)(80-15) \times 10^3$$



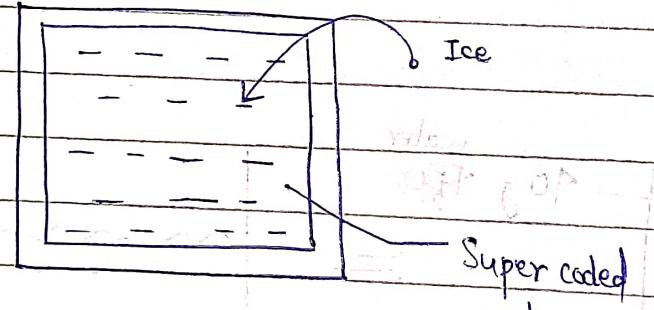
$$m = (1.12)(65) \times 10^3$$

$$(560)$$

 $\Rightarrow$ 

$$m = 130 \text{ g}$$

★ Q)



find fraction of  
water  
converted to  
ice.

$$\delta_{\text{ice}} = 0.5 \text{ cal/g } ^\circ\text{C}$$

$$\delta_w = 1 \text{ cal/g } ^\circ\text{C}$$

$$L_f = 80 \text{ cal/g}$$

A)



Whenever ice and water in eq.,

$$T_{\text{mix}} = 0^\circ\text{C}$$

Let  $m$  be mass of water. Let fraction  
' $f$ ' be converted to ice.

$$Q_{\text{loss}}(\text{by ice}) = (mf)(80)$$

$$Q_{\text{gain}}(\text{by ice \& water}) = (mf)(0+40) + m(1-f)(0+40)(1) \quad (1/2)$$

$$\Rightarrow m(80f) = m(20f + 40 - 40f)$$

$$\Rightarrow 4f = f + 2 - 2f \Rightarrow f = 2/5$$

### Graphs

Consider we are heating ice at  $-5^{\circ}\text{C}$ .

for  $-5^{\circ}\text{C} \leq T < 0^{\circ}\text{C}$ ,

$$Q = m s_{\text{ice}} (T + 5)$$

$$T = (-5) + \frac{Q}{m s_{\text{ice}}} \quad (\text{given by us})$$

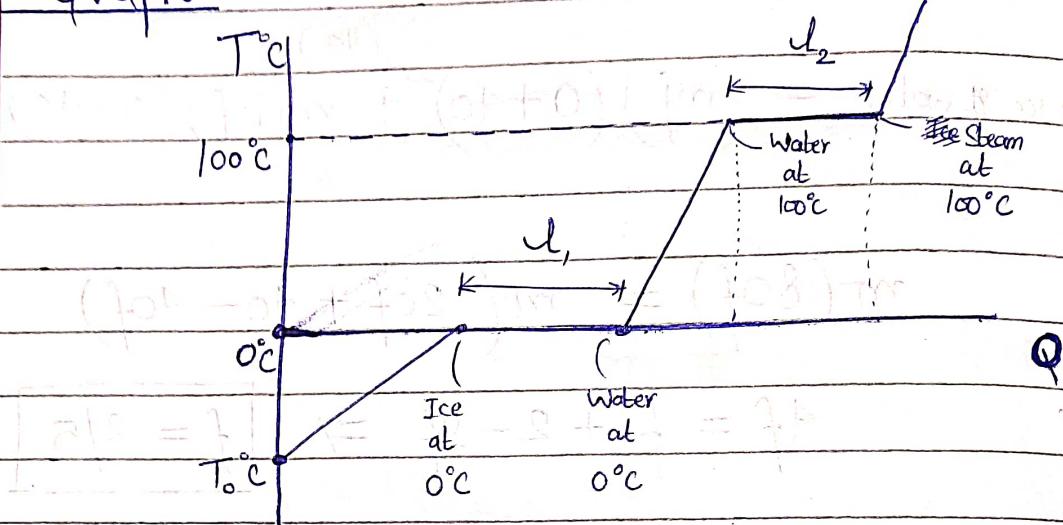
for  $T = 0^{\circ}\text{C}$ ,  
(Phase change)

$$Q \in [m s_{\text{ice}} \cdot 5, m s_{\text{ice}} \cdot 5 + mL_f]$$

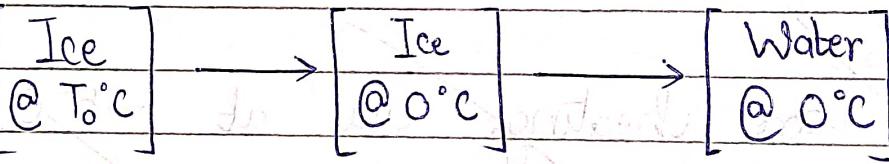
for  $T > 0^{\circ}\text{C}$ ,

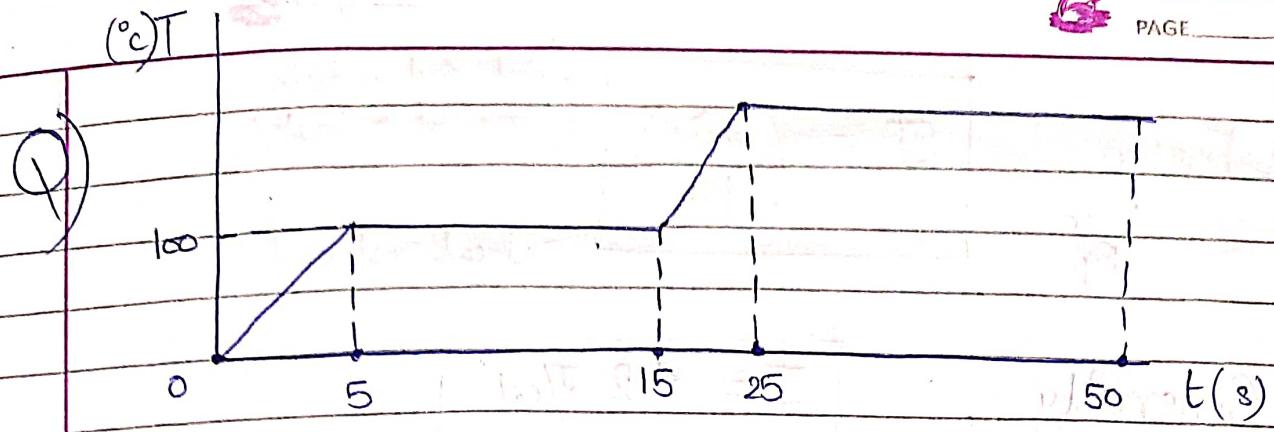
$$Q = m s_{\text{ice}} \cdot 5 + mL_f + m s_w T$$

## Graph



for ice starting at temp  $T_0$ ,





Rate of heating is const.  $m = 1 \text{ kg}$ .  $\delta_{\text{solid}} = 0.6 \frac{\text{kCal}}{\text{g}\cdot^\circ\text{C}}$

find  $L_f$  &  $L_v$ .

A) Let rate of heating be  $r$  in  $\text{kCal}/\text{min}$ .

$$\Rightarrow Q_1 = (r)(5) = ((1)) (0.6) (100) = 720 \quad (1)$$

$$\Rightarrow r = 12 \frac{\text{kCal}}{\text{s}}$$

Now,  $Q_f = (1)L_f = (12)(15-5) \Rightarrow L_f = 120 \frac{\text{kCal}}{\text{g}}$

$$Q_v = (1)L_v = (12)(50-25) \Rightarrow L_v = 300 \frac{\text{kCal}}{\text{g}}$$

## Mechanical Equivalent ON system

If some work done is fully converted to heat (NOT the other way around as converting heat fully to work is NOT possible), then amt. of heat produced is called its ~~work~~ mech. eq.

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done ON System

DATE  
PAGEMech.  
Eq.

$$J = \frac{W}{H}$$

(work that is fully converted to heat)

(heat released).

Generally,

$$J = 4.2 \text{ J/cal}$$

- Q) 100 m high water fall. On coming to ground 50% of work is converted into heat. Find rise in temp. of water.

A)  $W = mgh = (10^3) m \cdot (m/s^2) \cdot 100$

$$W_{(\text{converted fully to heat})} = 50\% \text{ of } (10^3) m \cdot m^2/s^2$$

$$= (5 \times 10^2) m \cdot m^2/s^2$$

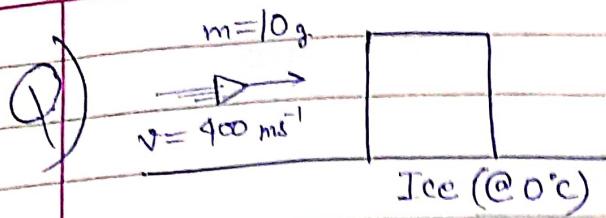
$$H = \frac{W_{(\text{fully converted to heat})}}{J} = \frac{(5 \times 10^2) m \cdot m^2/s^2}{4.2 \text{ J/cal}} =$$

Now,

$$Q = m s (\Delta T)$$

$$\Rightarrow (\Delta T)(m) \left( \frac{1 \text{ cal}}{g \cdot ^\circ C} \right) = \frac{(5 \times 10^2) m \cdot \text{cal}}{4.2 \text{ kg}}$$

$$\Delta T = \frac{5}{4.2} \cdot ^\circ C$$



$$(L_f = 80 \text{ cal/g})$$

Bullet gets embedded.  
Combination NOT moving.

Find amt. of ice melted

A)  $W = \Delta K = \frac{1}{2}mv^2 = \frac{1}{2}(10 \times 10^{-3})(4 \times 10^2)^2 = 8 \times 10^2 \text{ J}$   
(fully converted to heat)

$$H = \left( \frac{W}{J} \right) = \left( \frac{8 \times 10^2}{4.2} \right) \text{ cal}$$

Now, ~~is~~  $(m_{\text{melt}})(80 \text{ cal}) = \left( \frac{8 \times 10^2}{4.2} \right) \text{ cal}$

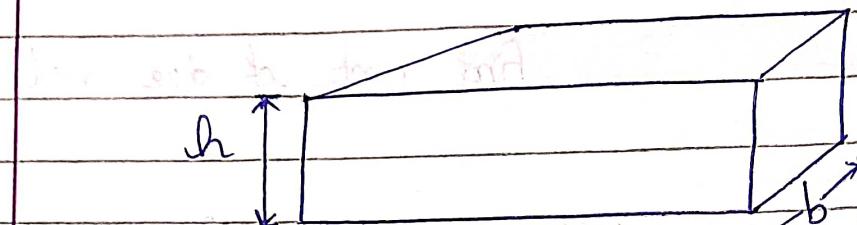
$$\Rightarrow m_{\text{melt}} = 50 \text{ g}$$

All done in this topic is useless.

Just conserve energy it include heat  
also, while taking care of units!

J is just a unit conversion factor.

## Thermal Expansion



If uniform heating & temp. of body inc. by  $(\Delta T)$ , then

$$l = l_0 (1 + \alpha \cdot \Delta T)$$

$$b = b_0 (1 + \alpha \cdot \Delta T)$$

$$h = h_0 (1 + \alpha \cdot \Delta T)$$

where

$$\alpha = \frac{1}{l_0} \left( \frac{dl}{dT} \right)$$

inc. in length of  
lm rad by  $1^\circ C$

where length is ANY length (linear measure)

$$\text{Now, } A = lb = l_0 b_0 (1 + \alpha \cdot \Delta T)^2$$

$$\Rightarrow A = l_0 b_0 (1 + 2\alpha \cdot \Delta T + \alpha^2 (\Delta T)^2)$$

$(6)$

$$\Rightarrow A = A_0 (1 + (2\alpha) \cdot \Delta T)$$

Similarly,

$$V = V_0 (1 + (\beta \alpha) \cdot \Delta T)$$

In general,

Linear measure	$L = L_0 (1 + \alpha \cdot \Delta T)$	Coeff. of Linear exp^n
Area measure	$A = A_0 (1 + \beta \cdot \Delta T)$	Coeff. of Area exp^n
Vol. measure	$V = V_0 (1 + \gamma \cdot \Delta T)$	Coeff. of Vol. exp^n

where

$$\alpha = \beta = \frac{\gamma}{3}$$

Q)

$$|TATD| = |TA|$$

$(L_1, \alpha_1)$

$(L_2, \alpha_2)$

On heating to produce same temp. diff,  
change in length is same.

Find rel^n b/w given glys.

A)  $\Delta L_1 = L_1 \alpha_1 \Delta T$  Given,  $\Delta L_1 = \Delta L_2$   
 $\Delta L_2 = L_2 \alpha_2 \Delta T$   $\Rightarrow L_1 \alpha_1 = L_2 \alpha_2$

## Change in Time in Pendulum Clock

$$T_L = 2\pi \sqrt{\frac{L}{g}} \Rightarrow T'_L = 2\pi \sqrt{\frac{L'}{g}}$$

Time period

$$\Rightarrow \left(\frac{T'_L}{T_L}\right) = \sqrt{\frac{L'}{L}}$$

$$\Rightarrow \left(\frac{T'_L}{T_L}\right) = \sqrt{K(1 + \alpha \cdot \Delta T)} \quad \text{temp. diff.}$$

$$= \sqrt{1 + \alpha \cdot \Delta T}$$

$$\Rightarrow T'_L = T_L \left(1 + \frac{\alpha \cdot \Delta T}{2}\right)$$

$$\Rightarrow \boxed{\Delta T_L = \frac{1}{2} \alpha T L \Delta T}$$

Change in  
Time period

Change in  
temp.

(Generally,  $T = 2s$ )



$$\left(\frac{\Delta T_L}{T_L}\right) = (\# \text{ s lost in a day}) / (\# \text{ s in a day}) = \frac{1}{2} \alpha \cdot \Delta T$$

Q) At  $5^\circ\text{C}$ , a pendulum clock gains 10s in a day. At  $15^\circ\text{C}$ , it loses 90s in a day. Find temp. at which it gives correct time.

A)

Let it give correct time at  $T^\circ\text{C}$ .



Gaining Time  $\leftrightarrow$  Time Period dec

Losing Time  $\leftrightarrow$  Time Period inc

$$(-10) = \frac{1}{2} \cdot \alpha \cdot T \cdot (T-5) \cdot (5-T)$$

$$(+20) = \frac{1}{2} \cdot \alpha \cdot T \cdot (T-15)$$

$$10 = \frac{1}{2} \cdot \alpha \cdot T \cdot (T-5)$$

$$20 = \frac{1}{2} \cdot \alpha \cdot T \cdot (15-T)$$

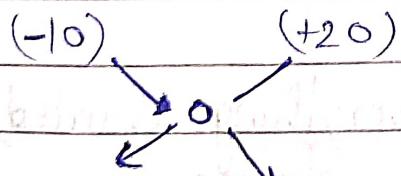
$$\frac{1}{2} = \frac{(T-5)}{(15-T)}$$

$$\Rightarrow T = 25/3 {}^\circ C$$

Alternative:

$$\Delta T_i : -10 \quad 0 \quad +20$$

$$T : 5 \quad T \quad 15$$



$$20 : 10 = (15-T) : (T-5) \Rightarrow T = 25/3 {}^\circ C$$

## Thermal Stress

If a ~~wall~~ <sup>rod</sup> fix b/w walls is heated its expansion is opposed by the wall  $\Rightarrow$  Thermal stress generated.

$$(T - T_0) \quad L = L_0 (1 + \alpha \cdot \Delta T)$$

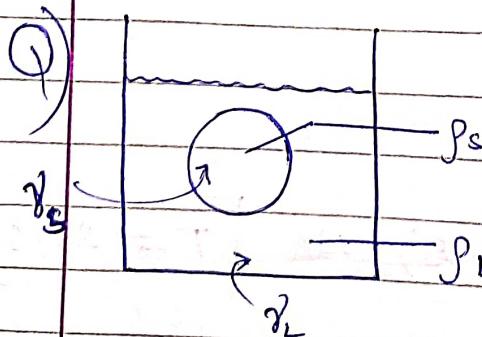
$$\Delta L = \alpha L_0 \cdot \Delta T$$

$$\left( \frac{\Delta L}{L_0} \right) = \alpha \Delta T$$

Now, Stress =  $\gamma$  (strain)



$$\boxed{\text{Stress} = \gamma \alpha \cdot \Delta T}$$



If everything heated, find change in buoyant force.  
(Assume full obj. submerged.)

A) \*

Liq. ALWAYS expand volumetrically.

$$V'_s = V_s (1 + \gamma_s \cdot \Delta T) \quad \text{et} \quad p'_s V'_s = p_s V_s$$

$$V'_L = V_L (1 + \gamma_L \cdot \Delta T) \quad \text{et} \quad p'_L V'_L = p_L V_L$$

(Vol. exp<sup>n</sup>)

(Mass. Const.)

$$\text{Now, } F_B = p_L V_s g \quad \text{et} \quad F'_B = p'_L V'_s g$$

$$\Rightarrow \Delta F_B = (p'_L V'_s - p_L V_s) g$$

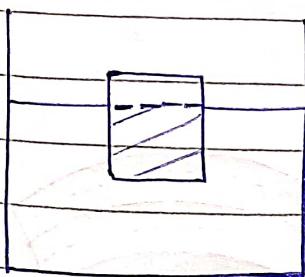
$$= (p_L V_s g) \left( \frac{p'_L \cdot V'_s}{p_L \cdot V_s} \right) - (p_L V_s g)$$

$$= (p_L V_s g) \left( \frac{(V'_s/V_s)}{(V'_L/V_L)} \right) - (p_L V_s g)$$

$$= (p_L V_s g) (1 + \gamma_s \cdot \Delta T) (1 + \gamma_L \cdot \Delta T) - (p_L V_s g)$$

$$\Rightarrow \boxed{\Delta F_B = (p_L V_s g) (\gamma_s - \gamma_L) (\Delta T)}$$

Q)



find change in fraction  
of solid submerged, if  
everything heated.

$$(p_s < p_L)$$

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A)  $V'_s = V_s (1 + \gamma_s \cdot \Delta T)$  &  $\rho'_s V'_s = \rho_s V_s$

$V'_L = V_L (1 + \gamma_L \cdot \Delta T)$  &  $\rho'_L V'_L = \rho_L V_L$

Now,  $\rho_L \cdot V_{\text{sub}} = \rho_s \cdot V_s$

$$\Rightarrow \left( \frac{\text{fraction of Solid}}{\text{Submerged}} \right) = \left( \frac{\rho_s}{\rho_L} \right)$$

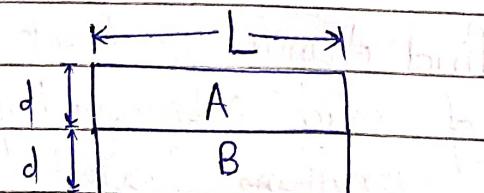
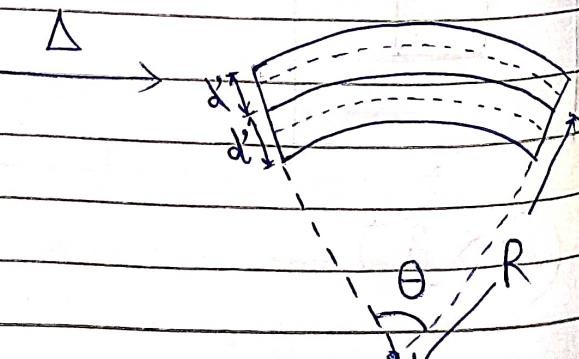
$$\Rightarrow \left( \frac{\text{New fraction of}}{\text{Solid submerged}} \right) = \left( \frac{\rho'_s}{\rho'_L} \right) = \left( \frac{\rho_s}{\rho_L} \right) \left( \frac{V_L / V_L}{V'_s / V_s} \right)$$

$$= \left( \frac{\rho_s}{\rho_L} \right) (1 + \gamma_L \cdot \Delta T) (1 + \gamma_s \cdot \Delta T)$$

$$= \left( \frac{\rho_s}{\rho_L} \right) (1 + (\gamma_L - \gamma_s) \Delta T)$$

$$\Rightarrow \boxed{\left( \frac{\text{Change in}}{\text{fraction}} \right) = \left( \frac{\rho_s}{\rho_L} \right) (\gamma_L - \gamma_s) (\Delta T)}$$

(1)

Find  $\theta$ .

A)  $L'_A = L_A (1 + \alpha_A \cdot \Delta T)$  &  $L'_B = L_B (1 + \alpha_B \cdot \Delta T)$ .

Now,  $(R + \frac{d}{2})\theta = L'_A$  &  $(R - \frac{d}{2})\theta = L'_B$

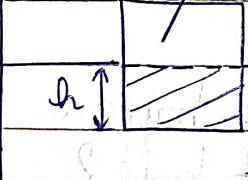
$$\Rightarrow d = \frac{L_A}{R \theta} \quad \left(\frac{d}{2}\right) = \left(\frac{L'_A - R}{\theta}\right) \quad \text{&} \quad \left(\frac{d}{2}\right) = \left(\frac{R - L'_B}{\theta}\right)$$

$$\Rightarrow \frac{L'_A + L'_B}{\theta} = 2R = \frac{L_A (1 + \alpha_A \cdot \Delta T)}{\theta} + \frac{L_B (1 + \alpha_B \cdot \Delta T)}{\theta}$$

$$\Rightarrow 2R = \left(\frac{L}{\theta}\right) (2 + (\alpha_A + \alpha_B) \cdot \Delta T)$$

$$\Rightarrow \boxed{\theta = \left(\frac{L}{2R}\right) (2 + (\alpha_A + \alpha_B) \cdot \Delta T)}$$

$$\textcircled{1} \quad \alpha_s$$



If on heating,  $h$   
(height of solid inside liq.)  
remains same, find  
rel'n b/w  $\alpha_s$  &  $r_s$

A) Let cross section area of solid be  $A_s$ .

$$\Rightarrow A'_s = A_s (1 + \alpha_s \cdot \Delta T)$$

Now,  $V' = V_s (1 + \gamma_L \cdot \Delta T)$  &  $\rho' V'_s = \rho_s V_s$

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PAGE

$$\text{Now, } mg = F_B = (F'_B) \cancel{(A_s + A_L)} = (A'_s) \cancel{(h)} P'_L g$$

$$\Rightarrow (A_s) \cancel{(h)} P_L g = (A'_s) \cancel{(h)} P'_L g$$

$$\Rightarrow \left( \frac{A'_s}{A_s} \right) \left( \frac{P_L}{P'_L} \right) = 1$$

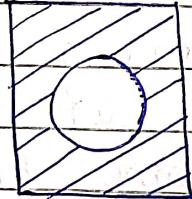
~~$$\Rightarrow (1 + 2\alpha_s \cdot \Delta T) \cancel{\left( \frac{V_i}{V_L} \right)} = \cancel{\left( \frac{V'_i}{V_L} \right)}$$~~

~~$$\Rightarrow (1 + 2\alpha_s \cdot \Delta T) = \cancel{\left( \frac{V'_i}{V_i} - 1 \right)} = \cancel{0.98}$$~~

$$\Rightarrow \left( \frac{A'_s}{A_s} \right) = \left( \frac{V'_i}{V_i} \right) \Rightarrow (1 + 2\alpha_s \cdot \Delta T) = (1 + \gamma_L \cdot \Delta T)$$

$$\Rightarrow \boxed{\gamma_L = 2\alpha_s}$$

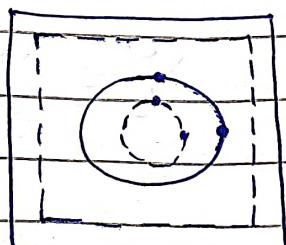
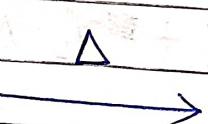
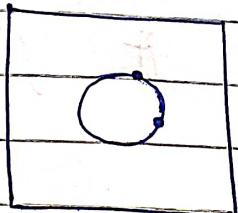
★ Q)



If heated uniformly, what happens to size of hole?

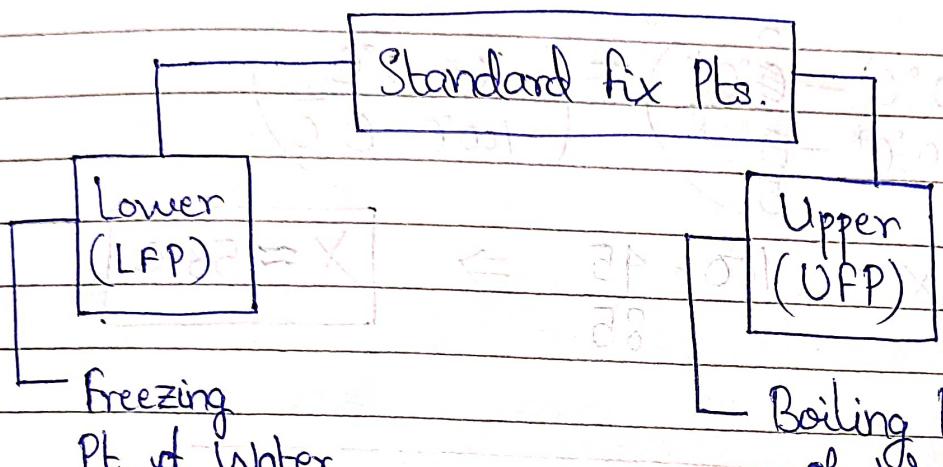
A)

On heating, avg. dist. b/w any 2 molecules inc. Consider 2 pts on circumference of hole.



Dist. b/w them should inc.  $\Rightarrow$  Radius inc.

## Thermometry (old)



### Temp. Scale

$$\frac{100}{(U.F.P - L.F.P)} = \left(\frac{C}{F}\right)$$

L.F.P      U.F.P

1) Celsius ( $^{\circ}\text{C}$ )	$0^{\circ}\text{C}$	$100^{\circ}\text{C}$
2) Farenheit ( $^{\circ}\text{F}$ )	$32^{\circ}\text{F}$	$212^{\circ}\text{C}$

$$\left[ \frac{(\text{Reading}) - (\text{LFP})}{(\text{UFP}) - (\text{LFP})} \right] = \left( \frac{C}{100} \right) = \left( \frac{F - 32}{180} \right)$$

↑

In general for  
any scale

(Rel<sup>n</sup> b/w):  
Temp. diff

$$\frac{1^{\circ}\text{C}}{5} = \frac{1^{\circ}\text{F}}{9}$$

Q) faulty Therm. LFP = 5°C. UFP = 90°C.  
If temp. of body on this scale is 50°C. find actual temp.

A) Let us denote faulty by °C\*

$$\left( \frac{50^{\circ}\text{C}^* - 0^{\circ}\text{C}^*}{90^{\circ}\text{C}^* - 0^{\circ}\text{C}^*} \right) = \left( \frac{x - 0^{\circ}\text{C}}{100^{\circ}\text{C} - 0^{\circ}\text{C}} \right)$$

$$\Rightarrow x = 100 \cdot \frac{45}{85} \Rightarrow x = 53^{\circ}\text{C}$$

★ for any ~~therm~~ prop (i.e. ∝ temp.) thermometric

$$(\text{Temp.}) = \left( \frac{x_{T^{\circ}\text{C}} - x_{0^{\circ}\text{C}}}{x_{100^{\circ}\text{C}} - x_{0^{\circ}\text{C}}} \right) \cdot 100$$

Where  $x_{T^{\circ}\text{C}}$  is value of therm. prop at  $T^{\circ}\text{C}$ .

Eg - Resistance, Pressure, Length, ...

# Thermometry (Modern)

Only 1 fix pt.

$$\text{(Triple Pt. of Water)} = 273.16 \text{ K}$$

$$\text{(Temp.)} = \left( \frac{P_{TK}}{P_{273.16K}} \right) \times 273.16 \text{ K}$$

Where  $P_{TK}$  is presre of gas taken  
in const. vol. therm. at  $T \text{ K}$ .

100

# Heat Transfer

Slow

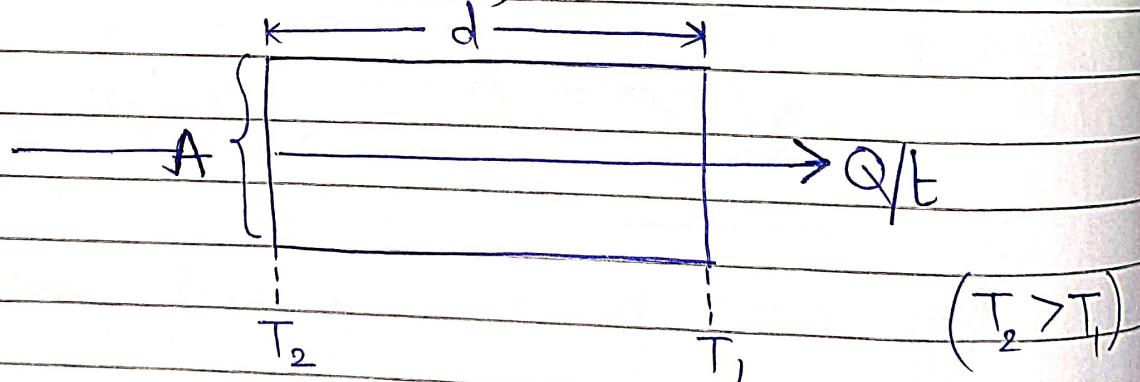
Conduction — In solids. Sometimes in liq.

Convection — In fluids.

Radiation — No medium req. Can happen in solid, liq., gas, vacuum

Fast

Conduction (in 1 D)



$$\text{Rate of Flow of Heat} = Q/T$$

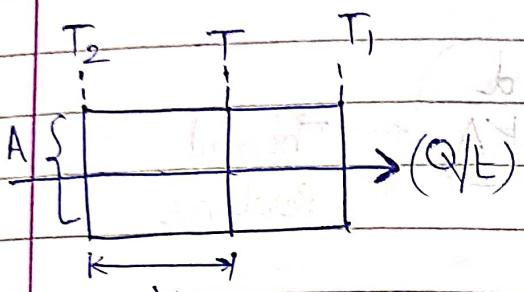
Thickness of Slab, in dir<sup>n</sup> of heat flow =  $d$

Area of Cross Section, normal to heat flow =  $A$

for Steady State,  
 (if  $K = \text{Const.}$ )  
 (if  $A = \text{Const.}$ )

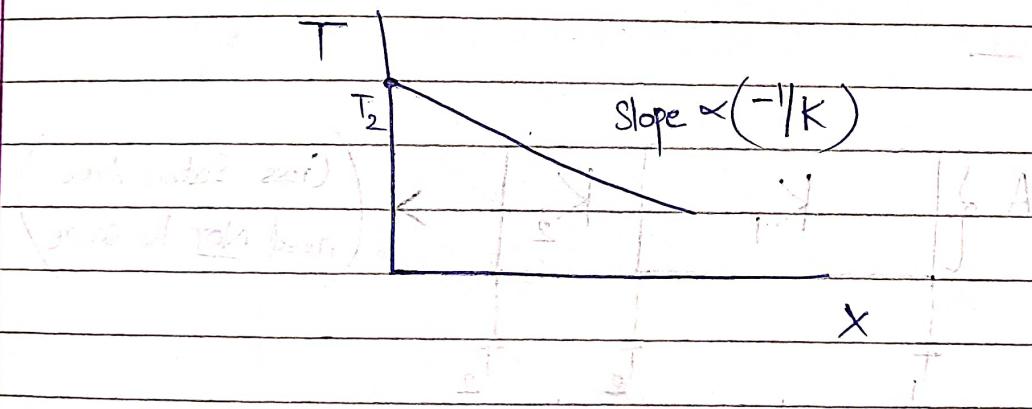
$$\left(\frac{Q}{t}\right) = \left(-\frac{KA(T_2 - T_1)}{d}\right)$$

Coeff. of Thermal Conductivity  
 (depends of material)

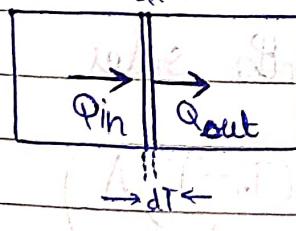


$$\left(\frac{Q}{t}\right) = \frac{KA(T_2 - T)}{x}$$

$$\Rightarrow T = T_2 - \left(\frac{1 \cdot Q}{KA} \right) x$$



In heat supply started,



$Q_{in} \neq Q_{out}$  as cross section is heating getting heated.

After some time,  $Q_{in} = Q_{out} \Rightarrow$  Steady State

for Steady State,

$$\left(\frac{dT}{dx}\right) = \left(-\frac{KA}{d}\right) = \left(\frac{Q}{t}\right)$$

(Diff. eqn for heat flow.)

Analogy  $\frac{Q}{t} \propto \frac{A}{L} \cdot i$

$$L \cdot i \iff Q/t$$

$$\text{Capacity of charge} \propto V \iff \Delta T$$

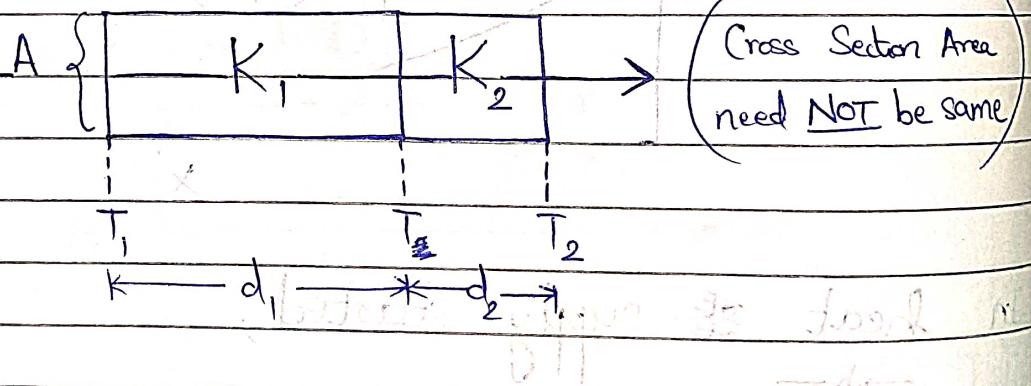
$$R \iff$$

$$\frac{d}{KA}$$

Thermal  
Resistance

### Combination of Slabs

#### i) Series -



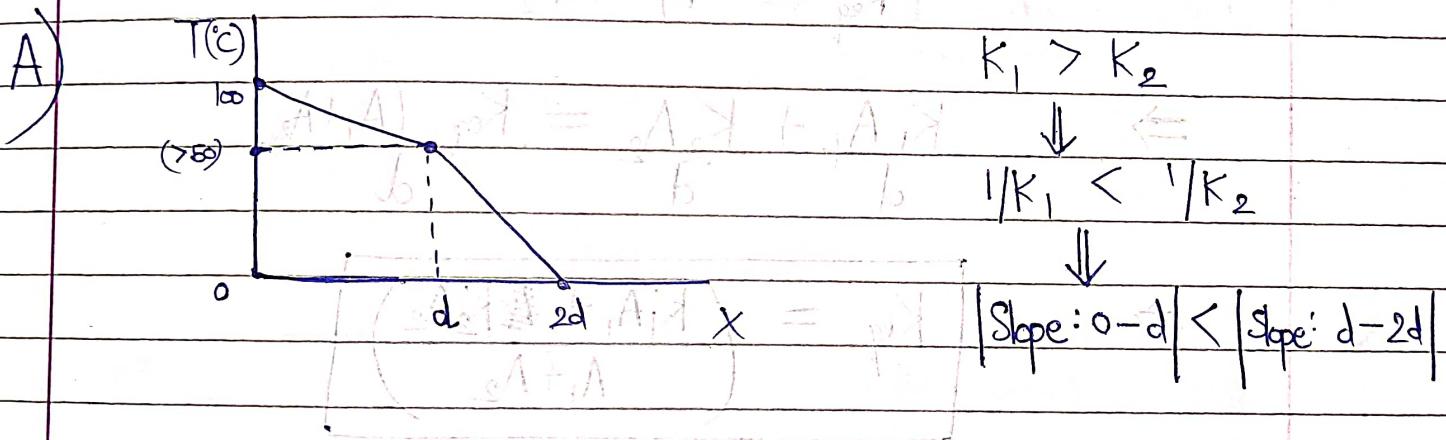
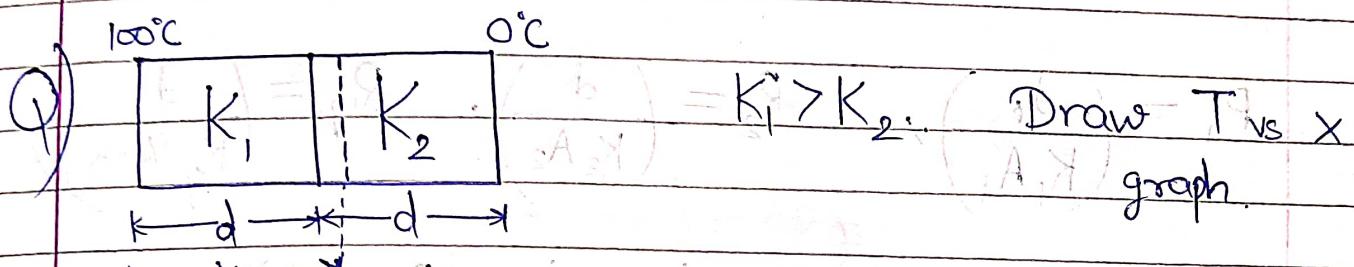
$Q/t$  across junction same on both sides,

$$\left(\frac{Q}{t}\right) = \left(\frac{K_1(T - T_1)A}{d_1}\right) = \left(\frac{K_2(T_2 - T)A}{d_2}\right)$$

- Q) If  $K_1/K_2 = 2$ ,  $d_1/d_2 = 1/2$ ,  $(T_1 - T_2) = 24$  K  
find  $(T_1 - T)$  junction

A)  $\frac{K_1(T-T_1)}{d_1} = \frac{K_2(T_2-T)}{d_2} \Rightarrow \frac{1}{T} = \left( \frac{24 - (T_1 - T)}{(T_1 - T)} \right)$

$$\Rightarrow \frac{96/5}{24/5} = \left( \frac{24 - (T_1 - T)}{(T_1 - T)} \right) \Rightarrow (T_1 - T) = 4.8 K$$



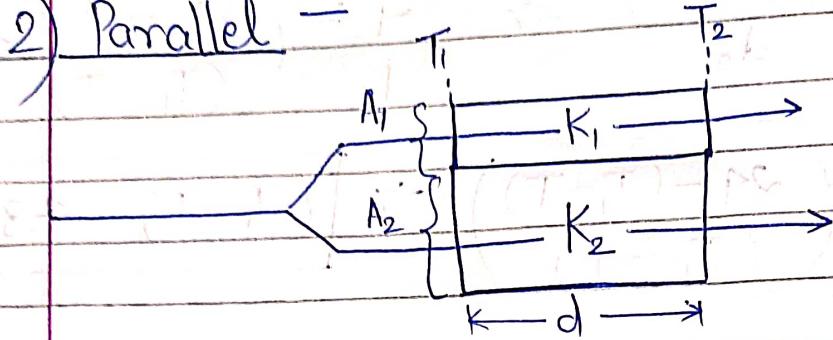
1.1) Equivalent  $K$ :

$$R_1 = \left( \frac{d_1}{K_1 A} \right) \quad R_2 = \left( \frac{d_2}{K_2 A} \right) \quad R_{eq} = \left( \frac{(d_1 + d_2)}{K_{eq}} \right)$$

Now,  $R_1 + R_2 = R_{eq}$  (Series)

$$\Rightarrow \left( \frac{d_1 + d_2}{K_{eq}} \right) = \frac{d_1}{K_1} + \frac{d_2}{K_2}$$

2) Parallel -



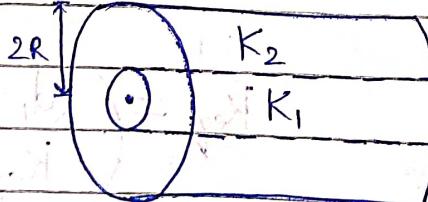
$$R_1 = \frac{d}{K_1 A_1}, \quad R_2 = \frac{d}{K_2 A_2}, \quad R_{\text{eq}} = \frac{d}{K_{\text{eq}}(A_1 + A_2)}$$

Now,  $\frac{1}{R_{\text{eq}}} = \frac{1}{R_1} + \frac{1}{R_2}$  (Parallel)

$$\Rightarrow \frac{K_1 A_1}{d} + \frac{K_2 A_2}{d} = K_{\text{eq}} \cdot \frac{(A_1 + A_2)}{d}$$

$$K_{\text{eq}} = \frac{(K_1 A_1 + K_2 A_2)}{A_1 + A_2}$$

(Q)



find \$K\_{\text{eq}}

A)

$$K_{\text{eq}} = \frac{K_1(\pi R^2) + K_2(4\pi R^2 - \pi R^2)}{4\pi R^2}$$

⇒

$$K_{\text{eq}} = \frac{(K_1 + 3K_2)}{4}$$

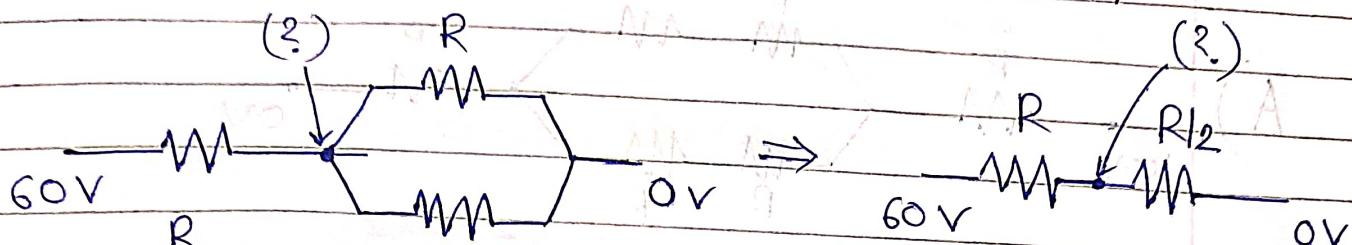
(Q) ~~60°C after 1A~~  
~~TE & heat~~

0°C

0°C

find temp. at junction. All rods identical.

A)



By KVL,  $60 - R(4i/R) = (?)$

$$i = \frac{60}{3R/2} \Rightarrow i = 40/R$$

 $\Rightarrow$ 

$$(?) = \frac{20}{R}$$

 $\Rightarrow$ 

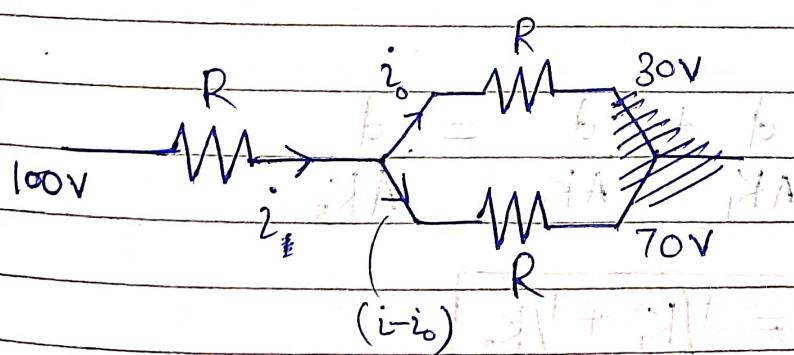
$$T_{\text{junction}} = 20^\circ\text{C}$$

(Q)

~~find for suff. for 60A~~  
~~100°C reading 3 min. time on~~  
~~(100/60)^2 = 70°C~~

find temp. at junction.  
All rods identical.

A)



$$100 - iR - i_o R = 30 \quad \text{---} ①$$

$$100 - iR - (i - i_o)R = 70$$

$$\Rightarrow 100 - 2iR + i_o R = 70 \quad \text{---} ②$$

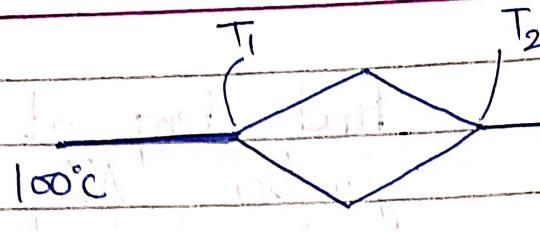
$$① + ② \Rightarrow 200 - 3iR = 100 \Rightarrow iR = 100/3$$

By KVL,  $100 - iR = V \Rightarrow V = 200/3 \Rightarrow$

$$200/3^\circ\text{C}$$

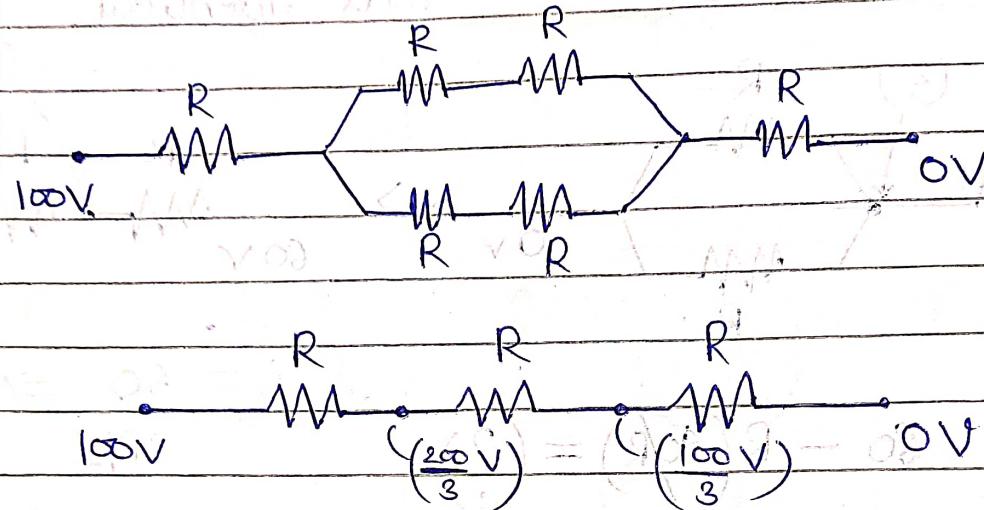
106

Q)



All rods identical  
find  $T_1$  &  $T_2$

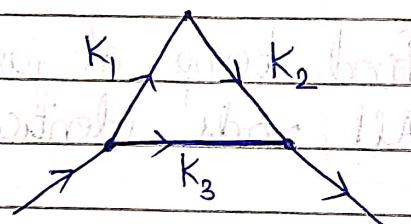
A)



$$\Rightarrow T_1 = \left(\frac{200}{3}\right)^\circ\text{C}$$

$$T_2 = \left(\frac{100}{3}\right)^\circ\text{C}$$

Q)



Rate of flow of heat  
is same in 2 paths.  
( $d, A$  = Same)

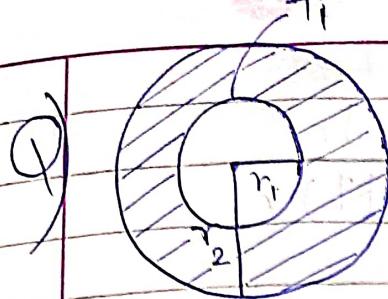
A)

$$R_1 = R_2 \Rightarrow \frac{d}{AK_1} + \frac{d}{AK_2} = \frac{d}{AK_3}$$

$$\Rightarrow \frac{1}{K_3} = \frac{1}{K_1} + \frac{1}{K_2}$$

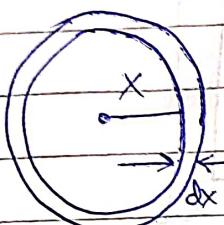


$\{T_1 > T_2\}$



A hollow sphere. Find  $Q/t$ .

A) Consider a shell of radius 'x'.



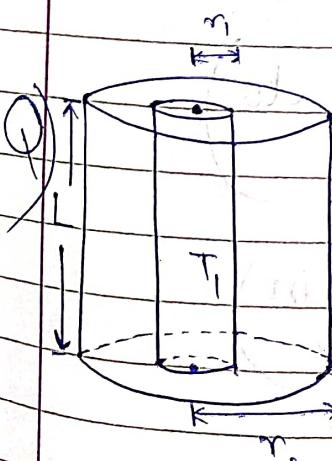
$$\int_{r_1}^x dR = \frac{dx}{K(4\pi x^2)}$$

$$\Rightarrow R = \int_{r_1}^x 1 \cdot x^2 dx$$

$$\Rightarrow R = \frac{1}{3} \cdot \left(\frac{1}{x}\right)^3 \Rightarrow R = \left(\frac{1}{4\pi k}\right) \left(\frac{1 - 1}{r_1 - r_2}\right)$$

Hence,

$$\frac{Q}{t} = \frac{(T_1 - T_2)(4\pi k)(r_1 r_2)}{(r_2 - r_1)}$$



$\{T_1 > T_2\}$

A hollow cylinder.

Find  $Q/t$

A) Consider a cylindrical shell with radius 'r' and length 'l'.

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A)

$$dR = \frac{dr}{K(2\pi r L)} \Rightarrow R = \int_{r_1}^{r_2} \frac{dr}{(2\pi K L) r}$$

$$\Rightarrow R = \left( \frac{1}{2\pi K L} \right) \ln \left( \frac{r_2}{r_1} \right)$$

Hence,

$$\left( \frac{Q}{t} \right) = \left[ \frac{(T_1 - T_2)(2\pi K L)}{\ln(r_2) - \ln(r_1)} \right]$$

(Const.)

400K

(NOT const.)

300K

1 kJ/kgK

Q)

(1 kg, ~~1 m²~~) = (m, s)

$\leftarrow 1 \text{ m} \rightarrow$

disc ( $K = \infty$ )

( $K = 40 \text{ W/mK}$ )

(Area of  
Cross Section) =  $1 \text{ m}^2$

find time taken to inc. disc's temp. by  $50^\circ \text{C}$ . Init temp. of disc. = 300K

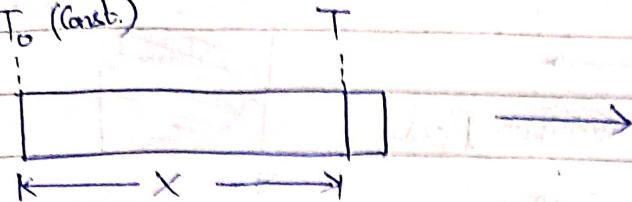
A)

$K_{\text{disc}} = \infty \Rightarrow$  ~~Allows heat to~~ pass easily

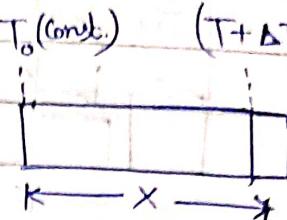
$\Rightarrow (dT/dx) = 0 \Rightarrow$  (Same temp in whole body)  
 $\left\{ (dT/dx) = \frac{Q/t}{K \cdot A} \right\}$

Consider  $(\Delta t)$  time period,

$T_0$  (const.)



$T_0$  (const.)  $\rightarrow (T + \Delta T)$



Now,

$$Q = (\Delta t) \left( \frac{KA}{x} \right) \frac{(T - T_0)}{ms} = ms \cdot (\Delta T)$$

Heat transferred in rod

Heat taken by disc.

$$\Rightarrow \frac{(\Delta T)}{\Delta t} = \left( \frac{KA}{x ms} \right) (T - T_0) (-1)$$

$\Rightarrow$

$$\frac{dT}{dt} = \left( \frac{KA}{x ms} \right) (T - T_0) (-1)$$

$$\Rightarrow \left( - \int_{T_0}^{T_1} \frac{dT}{T - T_0} \right) = \int_0^t \left( \frac{KA}{x ms} \right) dt$$

$\Rightarrow$

$$\boxed{\ln \left( \frac{T_1 - T_0}{T_2 - T_0} \right) = \left( \frac{KA}{x ms} \right) t}$$

In this  $Q$ ,  $T_0 = 400 \text{ K}$ ,  $T_1 = 300 \text{ K}$ ,  $T_2 = 350 \text{ K}$ ,

$K_r = 40 \text{ W/mK}$ ,  $x = 1 \text{ m}$ ,  $A = 1 \text{ m}^2$ ,

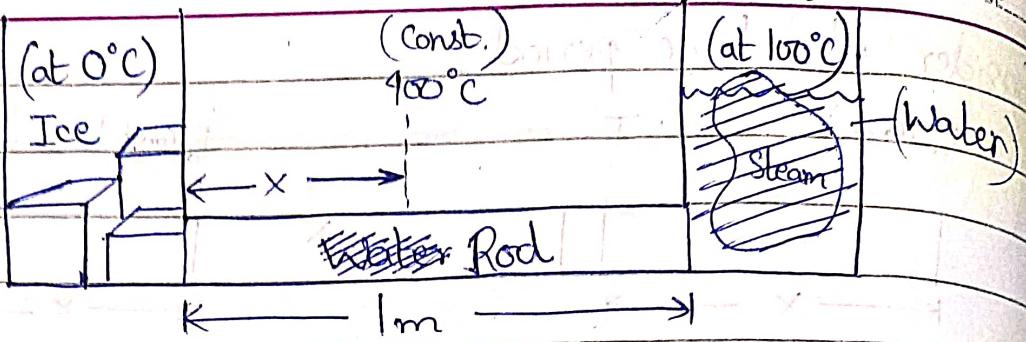
$m = 1 \text{ kg}$ ,  $c = 4 \text{ kJ/kg.K}$

$$\Rightarrow \left( -\ln \left( \frac{350 - 400}{300 - 400} \right) \right) = \left( \frac{40 \cdot 1}{1 \cdot 1 \cdot 4 \times 10^3} \right) t$$

$\Rightarrow$

$$\boxed{t = 69.8}$$

(Q)



Given,  $\left( \frac{\text{Rate of Melting}}{\text{of Ice}} \right) = \left( \frac{\text{Rate of forming}}{\text{of Steam}} \right)$

find  $x$  ( $L_f = 80 \text{ cal/g}$ ,  $L_v = 540 \text{ cal/g}$ )

A)  $\left( \frac{Q_{\text{ice}}}{t} \right) = L_f \left( \frac{m_{\text{ice formed}}}{t} \right) = (KA)(400)$

$\left( \frac{Q_{\text{water}}}{t} \right) = L_v \left( \frac{m_{\text{steam formed}}}{t} \right) = (KA)(300)$

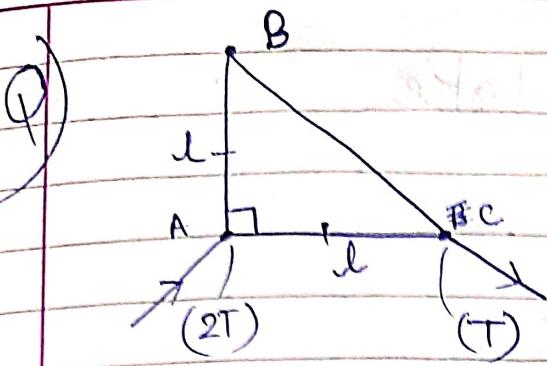
Given,  $\left( \frac{m_{\text{ice formed}}}{t} \right) = \left( \frac{m_{\text{steam formed}}}{t} \right)$

$\Rightarrow (KA)(400) = (KA)(300)$

$\Rightarrow \frac{x}{1-x} = \frac{400}{300} \cdot \frac{540}{80} = 9$

$\Rightarrow$

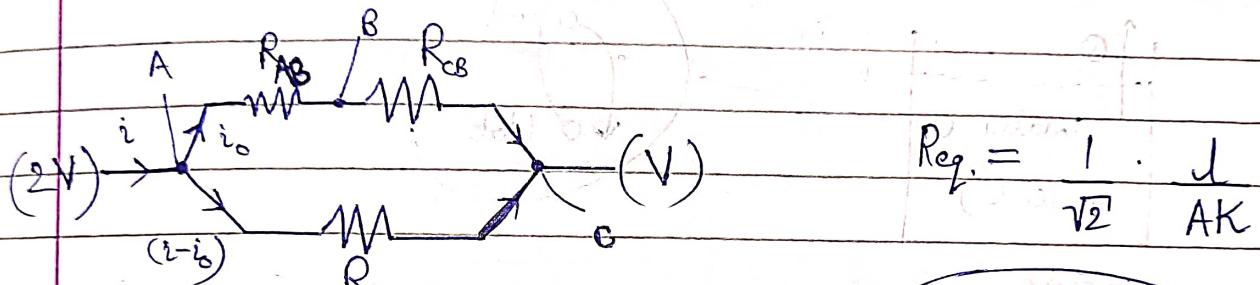
$x = 9/10$



Find  $T_B$

(All rods of same material)

$$A) R_{AC} = \frac{l}{AK}, \quad R_{BC} = \frac{l\sqrt{2}}{AK}, \quad R_{AB} = \frac{l}{AK}$$



$$R_{EQ} = \frac{l \cdot l}{\sqrt{2} AK}$$

$$i = \left( \frac{VAK}{l} \right) \sqrt{2}$$

$$\text{By KVL: } 2V - \left( \frac{l}{AK} \right) (\sqrt{2} + 1) \cdot i_0 = V$$

$$\Rightarrow i_0 = \left( \frac{VAK}{l} \right) (\sqrt{2} - 1)$$

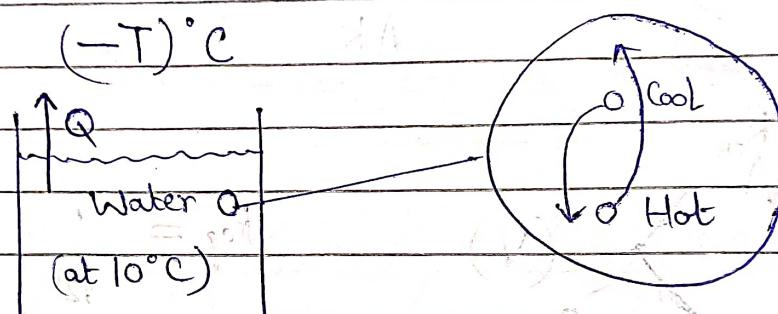
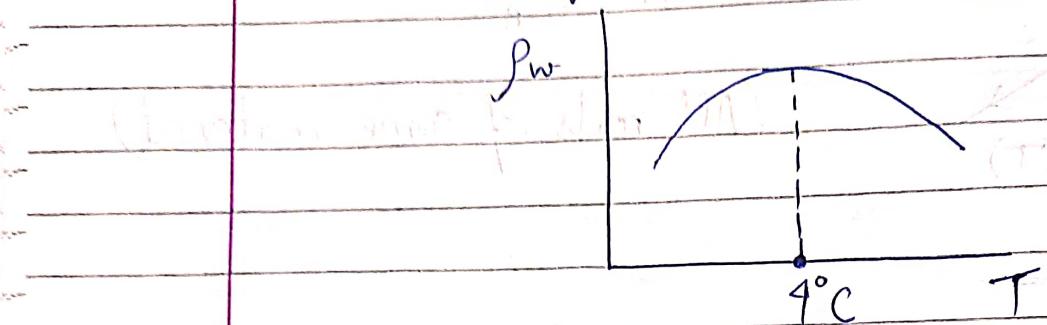
$$\text{Now, } 2V - i_0 \left( \frac{l}{AK} \right) = V_B \Rightarrow V_B = 2V - V(\sqrt{2} - 1)$$

$$V_B = (3 - \sqrt{2})V$$

Hence,

$$T_B = (3 - \sqrt{2})T$$

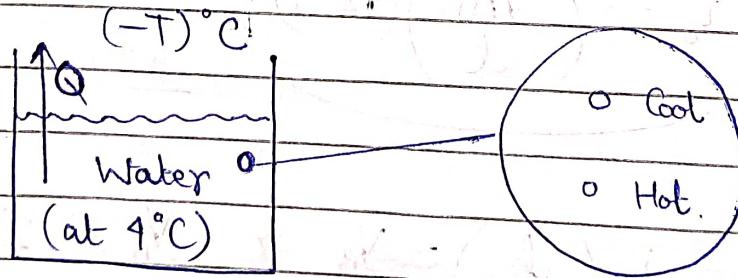
## Growth of Ice in a Lake



more

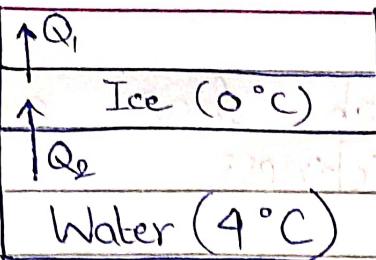
Cool dense particles come down  
dense particles come up.  $\Rightarrow$  (Uniform Cooling)

This happens till all water at  $4^{\circ}\text{C}$ .



After reaching  $4^{\circ}\text{C}$  temp., cooler molecules are less dense & remain at top.

Hence, top layer first reaches  $0^{\circ}\text{C}$ .

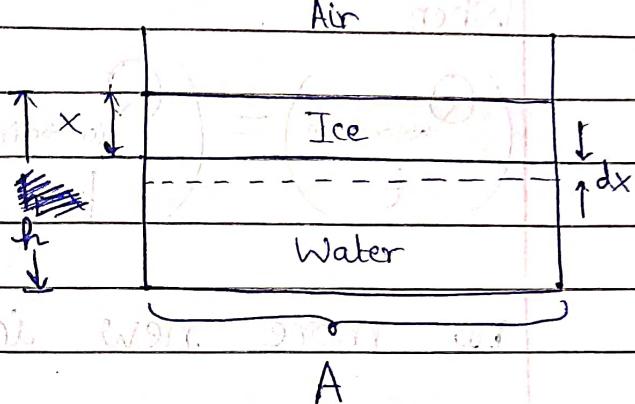


Ice conducts heat from water to air.

Let us find thickness of ice slab. Consider a slab of thickness 'x' is already present.

$$(M_{\text{new ice}}) = \rho_{\text{ice}} \cdot A \cdot dx$$

Now, heat absorbed by new ice



$$dQ = (M_{\text{new ice}}) L_f$$

$$= \rho_{\text{ice}} \cdot A \cdot L_f \cdot dx$$

$$\Rightarrow \left( \frac{dQ}{dt} \right) = \rho_{\text{ice}} \cdot A \cdot L_f \cdot \left( \frac{dx}{dt} \right)$$

By heat conduction thru ice,

$$\left( \frac{dQ}{dt} \right) = \frac{[0 - (-T)] (K_{\text{ice}} A)}{x}$$

$$\Rightarrow \left( \frac{dQ}{dt} \right) = (K_{\text{ice}} A) \left( \frac{T}{x} \right)$$

$$\text{Equating if } \rho_{\text{ice}} \cdot A \cdot L_f \cdot \left( \frac{dx}{dt} \right) = K_{\text{ice}} \cdot A \cdot \frac{T}{x}$$

$$\int_{x_i}^x dx = \int_0^t \left( \frac{K_{\text{ice}} \cdot T}{\rho_{\text{ice}} \cdot L_f} \right) dt$$

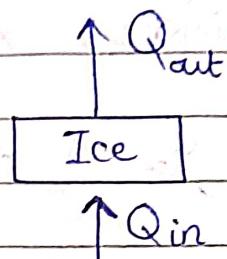
$$\Rightarrow t \propto (x_f^2 - x_i^2)$$

Now, the slab has a max. possible thickness.

Since from  $0^\circ\text{C} - 4^\circ\text{C}$ , water transfers heat via conduction, we have a water slab.

When,

$$\frac{(Q_{\text{water} \rightarrow \text{ice}})}{t} = \frac{(Q_{\text{ice} \rightarrow \text{air}})}{t}$$



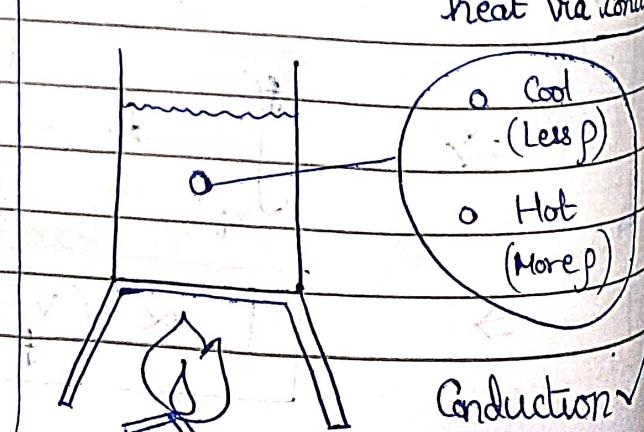
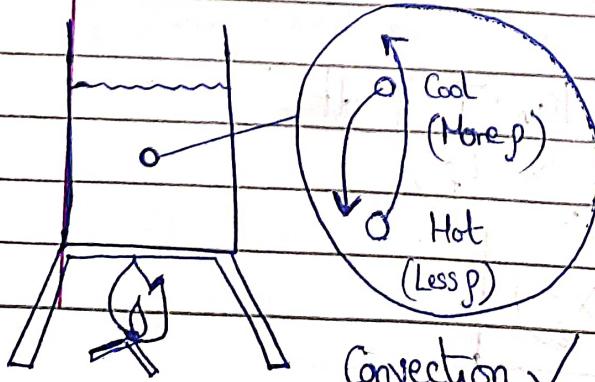
no more new ice forms.

$$\Rightarrow (K_w A) \left( \frac{4 - 0}{h - x} \right) = (K_{\text{ice}} A) \left( \frac{0 - (-T)}{x} \right)$$

$$\Rightarrow x_{\max.} = \left( \frac{T K_{\text{ice}} \cdot h}{T K_{\text{ice}} + 4 K_w} \right)$$

## Convection

Happens in a fluid whose ' $\rho$ ' DEC. on INC. T  
If a fluid's ' $\rho$ ' INC. with INC. T, then it transmits heat via conduction



# Radiation

## Black Body Radiation

(Surface Area)

D) Stefan's Law

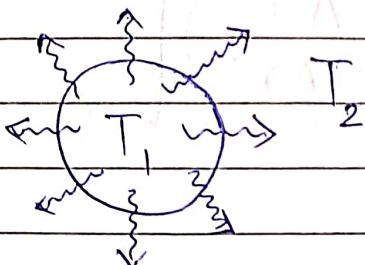
$$E = \sigma A T^4$$

(Emissive Power) =  $\frac{\text{Energy given off per sec.}}{\text{Const.}}$   
 (Abs. temp.)

$$\sigma = (\text{Stefan Boltzmann's Const.}) = (5.67 \times 10^{-8} \text{ W/m}^2\text{K}^4)$$

for a body with surface area A at temp. T.

Ex:

if  $T_1 < T_2$ ,

then ALSO body radiate heat.

It just happens that it absorbs more than it radiates.

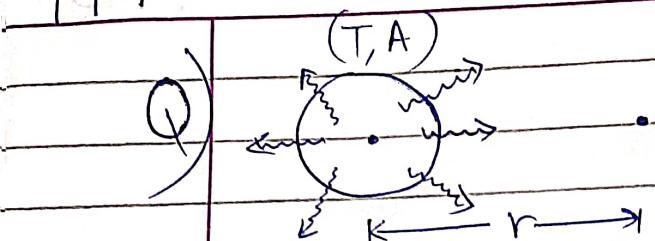
 $Q_{\text{radiated}}$ 

depends on

 $T_{\text{body}}$  $Q_{\text{absorbed}}$ 

depends on

 $T_{\text{sum}}$ Every body above  $0^{\circ}\text{K}$  radiates energy



find intensity at  $P$ .

A)  $(\text{Intensity}) = (\text{Energy per Area per Sec.})$

Radiation spreads out in a Sphere.

$$(Surface \text{ Area} \text{ of Spreading}) = 4\pi r^2$$

$$I = \frac{\sigma AT^4}{4\pi r^2}$$

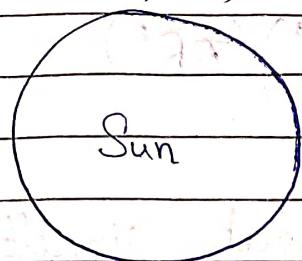


for spt / spherical source,  $I \propto \frac{1}{r^2}$

for cylindrical source,  $I \propto \frac{1}{r}$

for plane source,  $I \propto r^0$

(T, A)



Earth

Radius = R

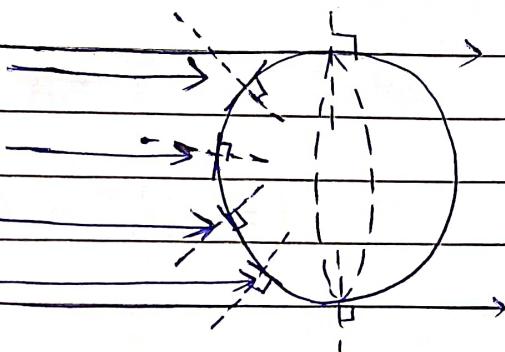
K

r

I

Since  $r \gg R$ , we can assume rays are coming  $\parallel$  it dist. of every pt. on Earth from Sun = r.

Now, ~~there~~ only 1 half receive it at every pt. ray are NOT normal to surface.



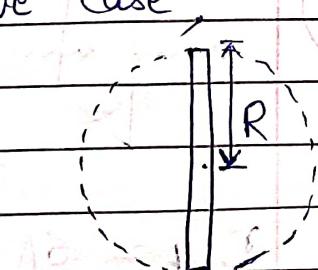
Only Normal component of rays is absorbed!

(i.e. comp. of rays normal to surface)

(Rate of Absorption) = (Intensity) (Effective Absorption Area)

By integration we get in above case

(Effective Area of Absorption) =  $\pi R^2$



i.e. area of disc thru centre of earth normal to line joining centres of Earth & Sun.

$$(\text{Rate of Absorption}) = \frac{6AT^4}{4\pi r^2} (\pi R^2)$$

Earth absorbs & emits heat simultaneously.

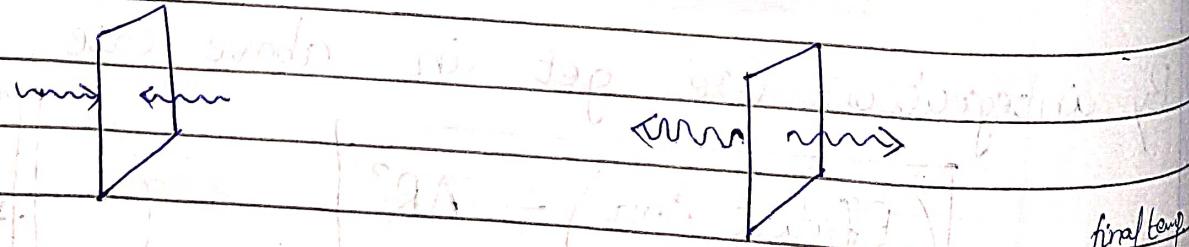
Now,  $(\text{Rate of Emission}) = \sigma (4\pi R^2) (T^4)$

Q)

Condition of Equilibrium: 3 identical plane sources.

Find temp. of middle plate with side  $(2A)$ .

A)



$$r_{\text{absorb}} = 6A(2T)^4 + 6A T^4$$

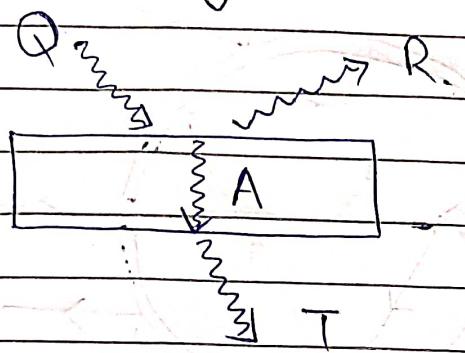
$$r_{\text{radiate}} = \sigma (2A) (T')^4$$

$$r_{\text{absorb}} = r_{\text{radiate}}$$

$$T' = T \left( \frac{17}{2} \right)^{1/4}$$

emit from both sides

When heat is given to a body,



It can be - reflected (R), absorbed (A) or transmitted (T)

Now

$$R/Q + A/Q + T/Q = 1$$

$$\Rightarrow R/Q + A/Q + T/Q = 1$$

reflectivity

## transmitivity

## Absorptivity

If body is Opaque  $\Leftrightarrow t=0$

If body is Blackbody  $\iff r=0, t=0$

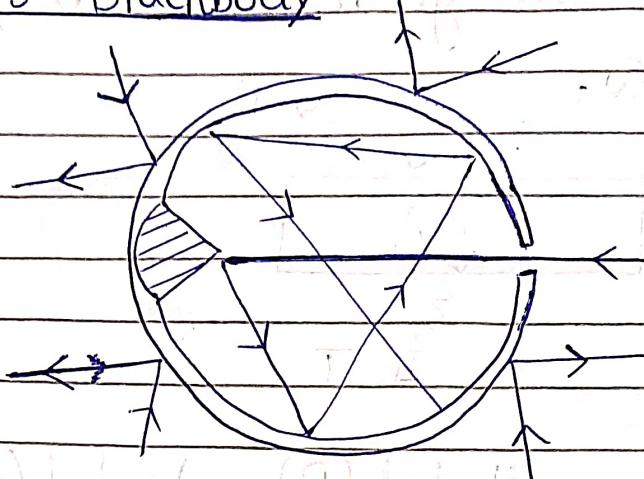
$$\Rightarrow a=1$$

Transmission  $\neq$  Radiation  $\neq$  Reflection

- Rays

  - 1) Transmission - ~~Radiation~~ passes w/o affecting body (passes right thru)
  - 2) Radiation - Remitting of rays after absorbing.
  - 3) Reflection - Rays jis medium se aati hain, usi main chali jati hain!

## Ferry's Blackbody



- 1) Blackened inside walls
- 2) Small entrance hole.
- 3) Reflector

Any light comes thru opening, gets reflected to sides. It remains inside. The sides absorb the light, one after another until whole light absorbed.

The opening behaves like a Blackbody!

The other surface are NOT blackbody as they may reflect light bouncing toward them.

$$\text{Emissivity } (\epsilon) = \frac{\text{Emissive power of body}}{\text{Emissive power of body, if it were blackbody}}$$

$$\begin{aligned} \text{Blackbody} &\iff \epsilon = 1 \\ \text{Whitebody} &\iff \epsilon = 0 \\ \text{Graybody} &\iff 0 < \epsilon < 1 \end{aligned}$$

for a graybody with temp ( $T$ ) & surface area ( $A$ ),

$$\boxed{(\text{Energy given out per sec}) = \epsilon \cdot \sigma A T^4}$$

If sum. at ' $T_0$ ' temp,

$$\boxed{(\text{Energy absorbed/sec}) = \epsilon \cdot \sigma A T_0^4}$$

$$\boxed{(\text{Net energy given out per sec}) = \epsilon \cdot \sigma A (T^4 - T_0^4)}$$

### Kirchoff's Law

$$\boxed{\frac{(\text{Emissive Power})}{(\text{Absorptive Power})} = \text{Const.}}$$

Since this is true for all obj's,

$$\frac{(\text{Emissive Power})}{(\text{Absorptive Power})}_{\text{Body}} = \frac{(\text{Emissive Power})}{(\text{Absorptive Power})}_{\text{Body, if it were blackbody}} = \frac{(\text{Emissive Power})}{(\text{Absorptive Power})}_{\text{if body were blackbody}}$$

$$\Rightarrow \frac{(\epsilon \cdot \sigma A T^4)}{a} = (6 A T^4) \Rightarrow \boxed{\epsilon = a}$$

$$\Rightarrow \boxed{\text{Emittance} = \text{Absorptivity}}$$

$\therefore$  Good absorbers  $\Rightarrow$  Good emitters

### Application —

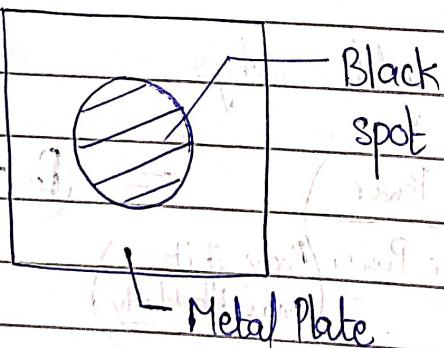
Rough  $\Rightarrow$  Fading Smooth  $\Rightarrow$  Non-fading

Since,  $r_{\text{rough}} < r_{\text{smooth}}$  It  $a+r=1$   
(Opaque body)

$$\Rightarrow q_{\text{rough}} > q_{\text{smooth}}$$

$\Rightarrow$  Rough body emits more than smooth body

2)



It is first heated in furnace, then taken in a dark room.

Then, Black spot will glow the brightest.

Red glass plate

It is heated in furnace it then taken in a dark room.

It appears Green.

Reason: Obj. looking red  $\Rightarrow$  It reflects Red most.  
 $\Rightarrow$  It absorbs Green most.

When kept in dark room, it reflects no light.  
 Only light ~~for~~  
<sup>from</sup> emmission only  $\Rightarrow$  Green (Antired)  
<sup>most detected</sup>

## Cooling

If  $T > T_0$ , it emits radiation.

$$\text{(Heat Loss)} = m\gamma(\Delta T) \quad (\text{if temp. of body red. by } |\Delta T| \text{ in time } \Delta t)$$

$$\Rightarrow \text{(Rate of Heat Loss)} = m\gamma \frac{(\Delta T)}{\Delta t} \quad (\text{as temp. dec})$$

$$= \epsilon \cdot A \sigma (T^4 - T_0^4)$$

$$\Rightarrow -m\gamma \frac{(dT)}{dt} = \epsilon \sigma A (T^4 - T_0^4) \quad (\text{Diff. Eqn of cooling})$$

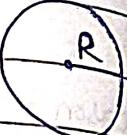
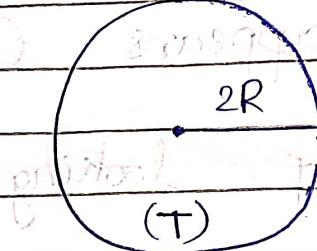
Observe,

$$\text{Rate of Cooling} \propto \frac{A}{m}$$

$$\text{Rate of Heat Loss} \propto A$$

(Q)

2 solid spheres are made of same material & have ~~an~~ identical surface finish.



- 1) find ratio of rate of heat loss
- 2) find ratio of rate of ~~heat~~ temp. loss (= cooling)

A) 1)  $(\text{Rate of Heat Loss}) \propto A \propto R^2$

 $\Rightarrow$ 

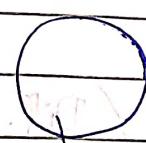
$$4 : 1$$

2)  $(\text{Rate of Cooling}) \propto \frac{A}{m} \propto \frac{R^2}{\rho R^3} \propto \frac{1}{R}$

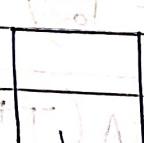
 $\Rightarrow$ 

$$1 : 2$$

(Q)

 $(m, \rho, s, T)$ 

Sphere



Cube

 $(m, \rho, s, T)$ 

Same surface finish.

find ratio of rate of temp. loss.

A)  $m$  same,  $\rho$  same  $\Rightarrow V$  same.

$$\Rightarrow \frac{4}{3} \pi R^3 = L^3$$

Now,  $(\text{Rate of Cooling}) \propto \left(\frac{A}{m}\right) = \left(\frac{A}{\rho V}\right)$

$$\Rightarrow \left(\frac{r_1}{r_2}\right) = \left(\frac{A_1}{A_2}\right) \left(\frac{V_2}{V_1}\right) = \left(\frac{4\pi R^2}{6L^2}\right) \left(\frac{\cancel{4/3\pi R^3}}{\cancel{4/3\pi R^3}}\right) \left(\frac{L^3}{\cancel{4\pi R^3}}\right)$$

$$\Rightarrow \left(\frac{r_1}{r_2}\right) = \left(\frac{L}{2R}\right) = \left(\frac{1}{2}\right) \left(\frac{4\pi}{3}\right)^{1/3}$$

$$\Rightarrow \boxed{(r_1/r_2) = (\sqrt[3]{16})}$$

Stefan's Law —  
of Cooling

$$\boxed{\left(\frac{dT}{dt}\right) \propto (T^4 - T_0^4)}$$

Newton's Law — If temp. diff. small,  
of Cooling

$$(T - T_0) \approx (T + T_0) \approx 2T_0 \quad \text{if } (T^2 + T_0^2) \approx 2T_0^2$$

$$\Rightarrow \left(\frac{dT}{dt}\right) \propto (T - T_0)(T + T_0)(T^2 + T_0^2) \approx (T - T_0)(2T_0)(2T_0^2)$$

$$\Rightarrow \left(\frac{dT}{dt}\right) \propto (T - T_0) \Rightarrow \boxed{\left(-\frac{dT}{dt}\right) = k(T - T_0)}$$

This is Newton's Law of Cooling

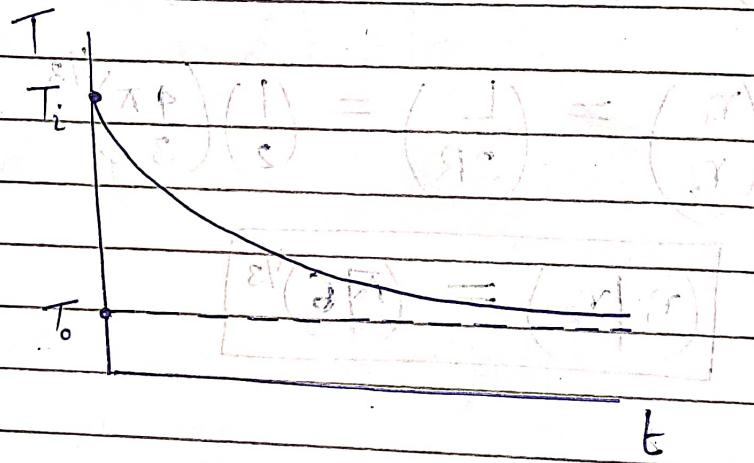
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$$\Rightarrow - \int_{T_i}^{T_f} \frac{dT}{T - T_0} = K \int_0^t dt$$

$$\Rightarrow \ln\left(\frac{T_f - T_0}{T_i - T_0}\right) = KT$$

$$\Rightarrow (T_f - T_0) = (T_i - T_0) e^{-KT}$$

for solving  $\frac{dT}{dt} = Q$ ,

$$\frac{dT}{dt} \approx \frac{(T_A - T_B)}{t}$$

$$T_m = \frac{(T_A + T_B)}{2}$$

$$\frac{dT}{dt} = k(T_m - T_0)$$

$$\frac{(T_A - T_B)}{t} = k(T_m - T_0)$$

~~A)~~  $70^\circ\text{C} \rightarrow 60^\circ\text{C} \Rightarrow t = 10 \text{ min.}$   
 $60^\circ\text{C} \rightarrow 50^\circ\text{C} \Rightarrow t = ?$

~~Q)~~  $T_{\text{urr}} = 25^\circ\text{C}$  & after  $10^\circ\text{C}$   $T_f = ?$

A) M1:

$$(T_f - T_0) = (T_i - T_0) e^{-kt}$$

$$\Rightarrow \frac{45}{35} = (e^k)^{(10)}$$

$$\text{et } \frac{35}{25} = (e^k)^t$$

$$\Rightarrow t = \frac{\ln(35) - \ln(25)}{\ln(45) - \ln(35)} \Rightarrow t \approx 13.38 \text{ min}$$

M2:  $\frac{T_A - T_B}{t} = k (T_m - T_0)$ 

$$\frac{(T_A - T_B)}{t} = k (T_m - T_0)$$

$$\Rightarrow \frac{10}{10} = k(65 - 25) \quad \text{et} \quad \frac{10}{10} = k(55 - 25)$$

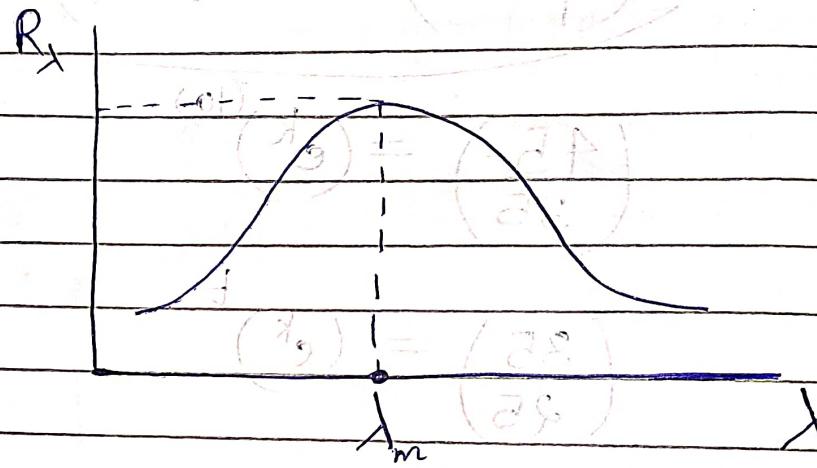
$$\Rightarrow \frac{t}{10} = \frac{65 - 25}{55 - 25} \Rightarrow t = \frac{40}{3} \text{ min}$$

## Plank's Distribution of Energy

An obj. above OK, emits radiations with all possible  $\lambda$ .

Some emitted in more qty., some in less.

$$(R_s)(T-sT) = (T-sT)$$



$R_s = \int R(\lambda) d\lambda = (R_{max})_{\lambda_m} - (R_{min})_{\lambda_m}$

$R_s$  - Radiation emitted with which  $R_s$  max.

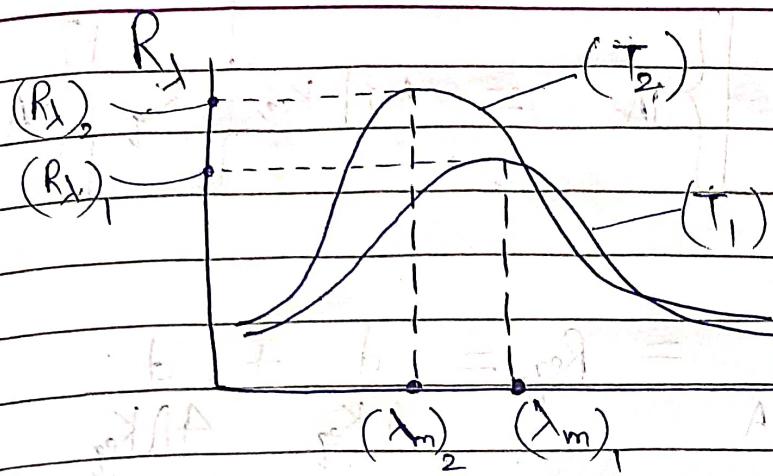


Area under  $R_s - \lambda$  graph gives (Energy emitted sec.)

## Wein's Disp. Law

$$T \cdot \lambda_m = b$$

$$\text{Wien's Const} = b = 2.9 \times 10^{-4} \text{ K} \cdot \text{m}^3$$



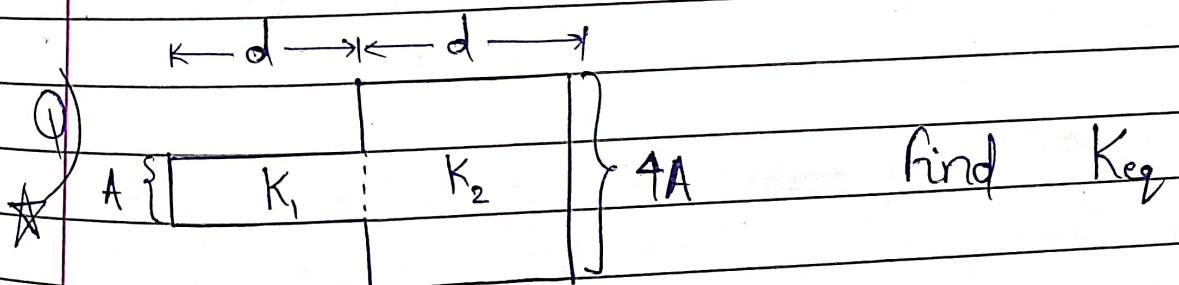
$(R_x)_2 > (R_x)_1 \Rightarrow$  Higher Peak

$(\lambda_m)_2 < (\lambda_m)_1 \Rightarrow$  Peak comes Earliest

$T_2 > T_1 \Rightarrow$  Temp. Higher.

Now,  $E \propto T^4$

$$\Rightarrow \boxed{(\text{Area under curve}) \propto T^4}$$



A) We will have problem if we directly apply  $R_{eq}$  as we would need  $A_{eq}$ . (with)

Instead use defn of  $K_{eq}$  Replace  $K_i$   ~~$K_{eq}$~~   $K_{eq}$  everywhere!

13)

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$$\text{A) } A \left\{ \begin{array}{c|c|c} K_1 & & \\ \hline & K_2 & \end{array} \right\} q_A \equiv A \left\{ \begin{array}{c|c|c} K_{eq} & & \\ \hline & K_{eq} & \end{array} \right\} A$$

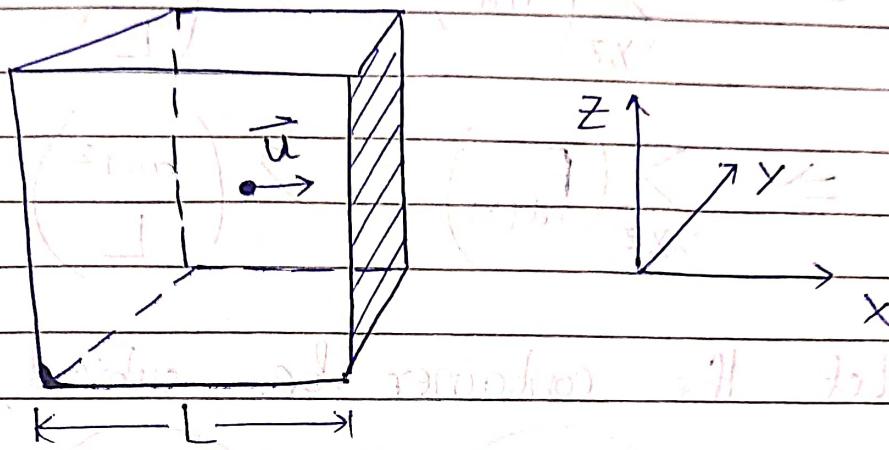
$\xrightarrow{\text{for } K_{eq}}$

$$\frac{d}{K_1 A} + \frac{d}{4K_2 A} = R_{eq} = \frac{d}{A K_{eq}} + \frac{d}{4A K_{eq}}$$

 $\Rightarrow$ 

$$K_{eq} = \left( \frac{5}{4(\frac{1}{K_1} + \frac{1}{4K_2})} \right)$$

# K T G.



We assume (all) ~~not~~ collisions are perfectly elastic.

$$\text{Now, } \Delta p_x = 2m u_x$$

$$\left[ \text{Time b/w 2 collisions (successive)} \right] = \frac{2L}{u_x}$$

$$\left( \text{Rate of } \Delta p_x \text{ w.r.t. Time} \right) = \frac{\Delta p_x}{\left( \text{Time b/w 2 collisions} \right)}$$

$$\Rightarrow \left( \text{force by wall on particle} \right) = \frac{(2m u_x)}{(2L/u_x)}$$

$$\Rightarrow \left( \text{Force on wall by particle} \right) = \frac{m u_x^2}{L}$$

$$\Rightarrow \left( \text{by } 1 \text{ particle} \right) F_x = \frac{(m u_x^2)}{L}$$

$$F_{\text{net}(x)} = \sum \left( \frac{mu_x^2}{L} \right)$$

$$\Rightarrow \sum_{x,y,z} (F_{\text{net}(x)}) = \sum \left( m \left( u_x^2 + u_y^2 + u_z^2 \right) \right)$$

$$\Rightarrow \sum_{x,y,z} (F_{\text{net}(x)}) = \sum \left( \frac{mu^2}{L} \right)$$

Let the container be cubical,

$$\Rightarrow V = L^3$$

$$A = L^2$$

$$\Rightarrow \sum_{x,y,z} \left( \frac{F_{\text{net}(x)}}{L^2} \right) = \sum \left( \frac{mu^2}{L^3} \right)$$

$$\Rightarrow P_x = \frac{(m \cdot n)}{L^3} \sum \left( \frac{u^2}{n} \right)$$

$$\Rightarrow (P_x + P_y + P_z) = \frac{(m_T)}{V} (u^2)$$

In absence of gravity,

$$P_x = P_y = P_z = P$$

(called Total  $P$ )

$$\Rightarrow 3P = \rho (\bar{u^2})$$

$$\Rightarrow P = \frac{1}{3} \rho (\bar{u^2})$$

Also,  $PV = \frac{1}{3} m \frac{(\bar{u^2})}{T}$

$$= \frac{2}{3} \sum \left( \frac{1}{2} mu^2 \right)$$

$$\Rightarrow PV = \frac{2}{3} (KE_{\text{gas}})$$

Now, we know  $P = \frac{RT}{M}$  (Molar mass of gas)

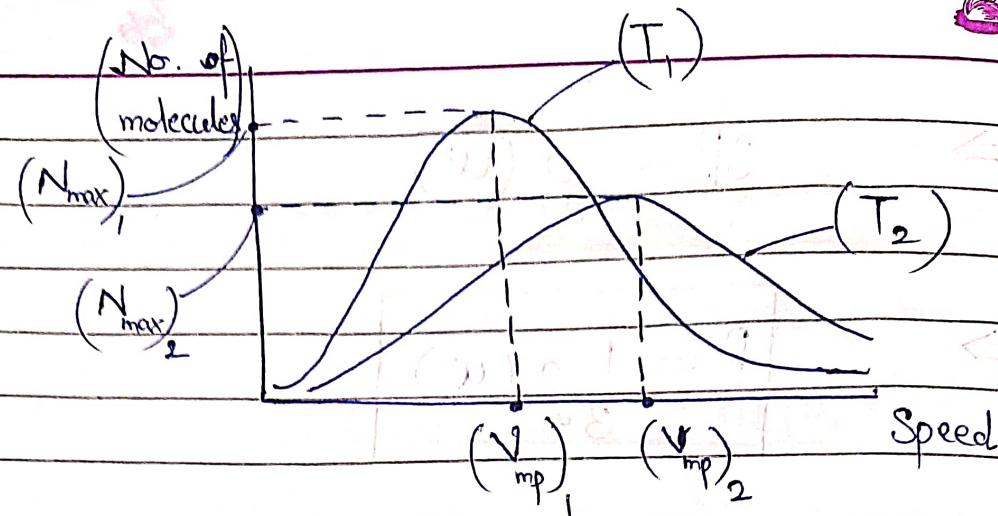
$$\Rightarrow \left( \frac{RT}{M} \right) = \frac{1}{3} (\bar{u^2})$$

$$\Rightarrow V_{\text{rms}} = \sqrt{\frac{\sum u^2}{n}} = \sqrt{\frac{3RT}{M}}$$

$$\text{Avg. vel.} = 0$$

$$\text{Avg. speed} = \sqrt{\frac{8RT}{\pi M}}$$

$$\text{(Most. Probable Speed)} = \sqrt{\frac{2RT}{M}}$$



$$N_{\max} \propto \left(\frac{1}{T}\right)$$

$$\{T_1 < T_2\}$$

### Degree of freedom

No. of independent ways in which a molecule can have energy.

Translational  $\rightarrow$  Max. (3)

Rotational  $\rightarrow$  Max. (3)

Vibrational  $\rightarrow$  Max. (2)

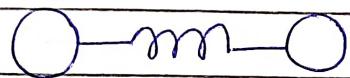
★ We see rotational degree of freedom about axis, about which system is Non ZERO.

MoI

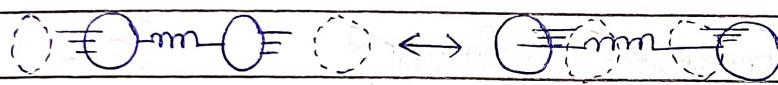
Atomicity	Trans.	Rot.	Dof
Mono.	3	0	3
Di.	3	2	5
Tri (Linear)	3	2	5
Tri (Non-linear)	3	3	6

### Vibrational Energy =

Analogy :



2 masses attached  
with a spring to  
each other



It is generally NOT comparable to other energies

Obviously, for monoatomic molecule it doesn't exist.

~~Consider~~ Consider vibrational energy in Dof only.  
when temp. high.

And ALWAYS,

$$\text{Dof} + = 2$$

if vib. dof. are inc.

3:

## Law of Equipartition of Energy

Energy associated with each dof is

$$\frac{1}{2} k_b T$$

$$k_b = \text{Boltzmann Const.} = \left( \frac{R}{N_A} \right)$$

Atomicity	Energy of 1 molecule	Energy of 1 mol.	$C_v m = \left( \frac{dU}{dt} \right)$
			(Internal Energy $U_{\text{gas}}$ )
Mono:	$\frac{3}{2} k_b T$	$\frac{3}{2} R T$	$\frac{3}{2} R$

Now,

$\star$  (KE of gas)  $=$  Int. Energy

$$U_{\text{gas}} = (f) \left( \frac{1}{2} R T \right)$$

★ Use Chem. wale formula. Just add ② to Dof. when high temp is there. Study from Chem. notes.

When ~~mix.~~ mix. of gases,

$$U_{\text{mix.}} = \sum (U_{\text{gas}})$$

Q) 3 mol O<sub>2</sub> + 2 mol He. Find ratio of avg. KE.

A)  $U_{O_2} = (5) \left( \frac{1}{2} RT \right) (3)$  X

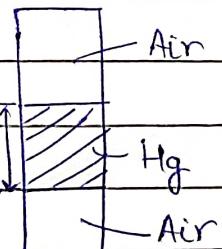
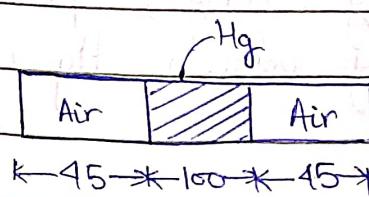
★ When avg. mentioned, we need (Energy Mol.)

$$\frac{U_{O_2}}{\text{mol}} = (5) \left( \frac{1}{2} RT \right) \quad \frac{U_{He}}{\text{mol}} = (3) \left( \frac{1}{2} RT \right)$$

$$\Rightarrow \text{Reg.} = 5/3$$

### Ideal Gas Eqn

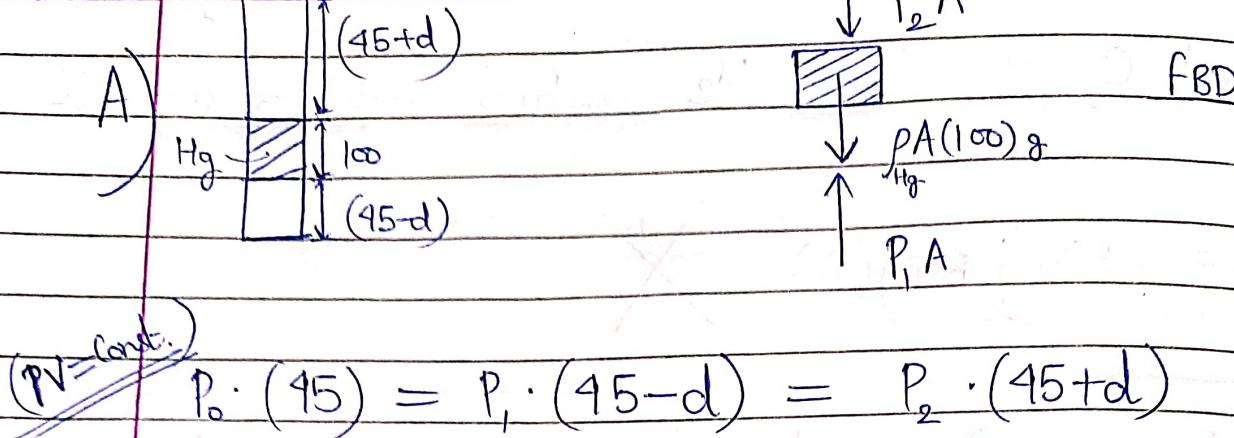
$$PV = nRT$$



Air init. at P<sub>0</sub>. find dist. moved by Hg after tube is made vertical.

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A)

~~(P1 = const.)~~

Now,

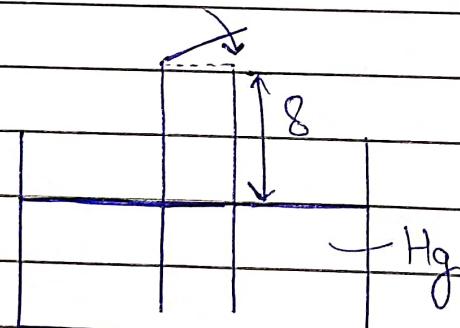
$$P_2 A + \rho_{Hg} A (100) g = P_1 A$$

 $\Rightarrow$ 

$$\rho_{Hg} (100) + \frac{(45 P_0)}{45+d} = \frac{(45 P_0)}{45-d}$$

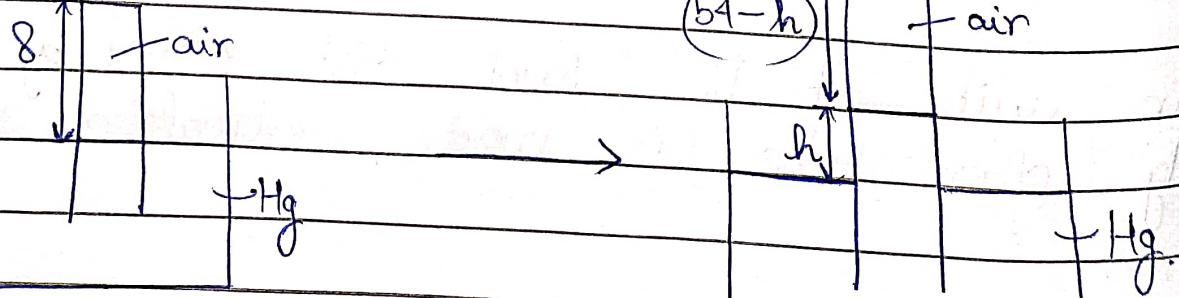
Now solve to find 'd'.

Q)



Top of tube is closed  
if tube is raised  
further by 46 cm.  
Assuming tube is  
very long, find  
height by which Hg rises.

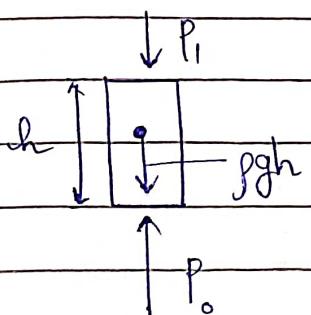
A)



Since, at first, vol. of air  $\uparrow \Rightarrow P \downarrow$  (less than  $P_0$ )

$\Rightarrow$  Due to  $P$  diff., Hg rises up.

Now,



$$P_1 + \rho gh = P_0$$

~~Also,~~  $P_1 \cdot (54 - h) = P_0 \cdot 8 \Rightarrow P_1 = \frac{8P_0}{54 - h}$

$$\Rightarrow \left( \frac{8P_0}{54 - h} \right) + \rho gh = P_0$$

In terms of Hg cm,  $\left( \frac{8 \cdot 76}{54 - h} \right) + h = 76$

$$\Rightarrow 8 \cdot 76 + (54h - h^2) = (76 \cdot 54 - 76h)$$

$$\Rightarrow h^2 - 130h + 76 \cancel{54} = 0$$
  
$$76 \cdot 46$$

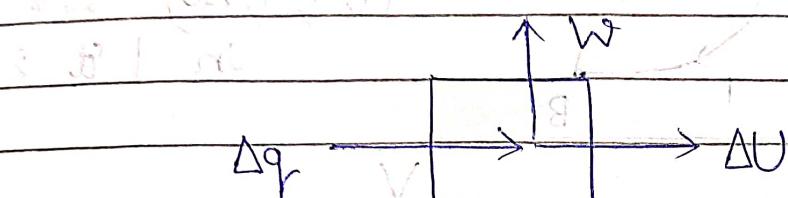
$$\Rightarrow h^2 - 130h + 38 \cdot 92 = 0$$

$$\Rightarrow (h - 38)(h - 92) = 0$$

$$\Rightarrow h = 38 \text{ cm}$$

# Thermodynamics

## 1st Law of Therm.



To remember use:

$$\Delta q_p = \Delta U + W$$

$$\Delta U = \Delta q + (-W)$$

Sign Convention:

(Heat given To system)  $\Leftrightarrow \Delta q > 0$

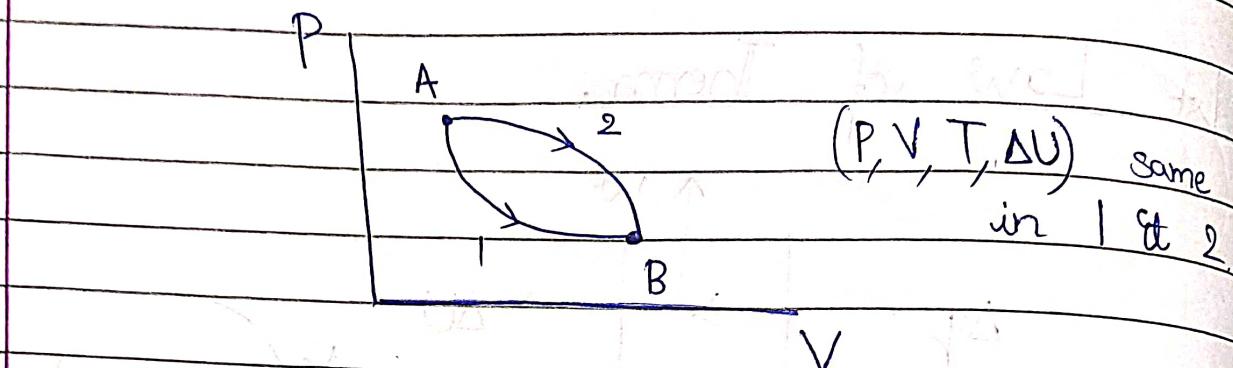
(Heat given By system)  $\Leftrightarrow \Delta q < 0$

(Expansion) (Work done By Gas)  $\Leftrightarrow W > 0$

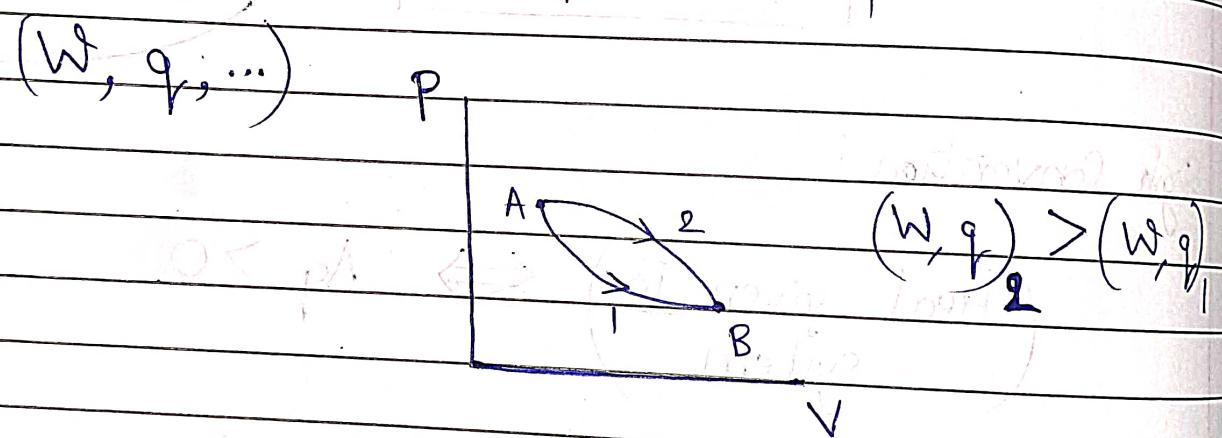
(Compression) (Work done ON Gas)  $\Leftrightarrow W < 0$

★ Study therm. chap. (before Carnot stuff) from Chem. notes. It put  $W \rightarrow (-W)$  in all formulae.

State fx<sup>n</sup>: Value depends only on initial & final pt. Doesn't depend on ~~path~~ path.



Path fx<sup>n</sup>: Value depends on path taken



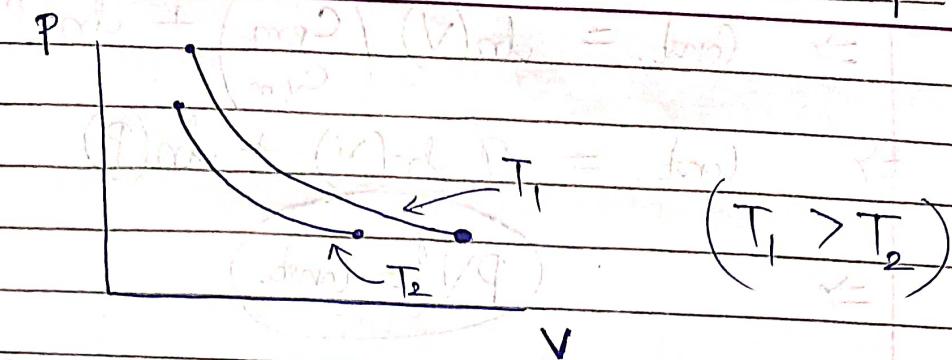
### Various Processes

1) Isothermal ( $iT$ ) -  $T$  is Const.

$$\Rightarrow \Delta U = 0$$

$$W = q$$

$$PV = \text{Const.}$$



$$W = (nRT) \ln \left( \frac{V_2}{V_1} \right)$$

2) Adiabatic (AdB) -

$$q = 0$$

$$\Rightarrow W + \Delta U = 0$$

Now,

$$PV^\gamma = \text{Const.}$$

(only if  $C_{Vm}$  independent of temp.)

Derivation:

$$q = W + \Delta U \approx$$

$$\Rightarrow 0 = nC_{Vm} \cdot dT + PdV$$

$$\Rightarrow 0 = nC_{Vm} \left( PdV + VdP \right) + PdV$$

$$\Rightarrow 0 = P \left( \frac{C_{Vm}}{R} + 1 \right) dV + V \left( \frac{C_{Vm}}{R} \right) dP$$

$$\Rightarrow 0 = \left( \frac{1}{V} \right) \left( \frac{C_{Vm}}{R} \right) dV + \left( \frac{1}{P} \right) \left( \frac{C_{Vm}}{R} \right) dP =$$

$$\Rightarrow \text{Const.} = \left( \frac{C_{Vm}}{R} \right) \ln(V) + \left( \frac{C_{Vm}}{R} \right) \ln(P) \approx$$

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$$\Rightarrow \text{Const} = \ln(V) \left( \frac{C_{p,m}}{C_{v,m}} \right) + \ln(P)$$

$$\Rightarrow \text{Const} = \gamma \ln(V) + \ln(P)$$

 $\Rightarrow$ 

$$PV^\gamma = \text{Const}$$

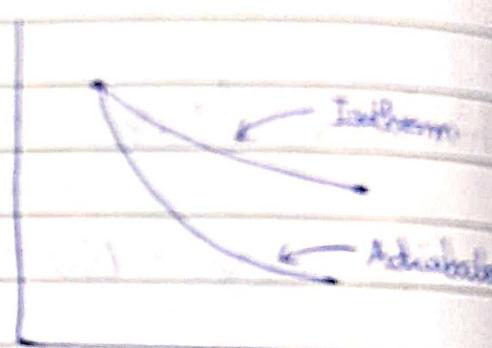
Now,

$$TV^{(\gamma=1)} = \text{Const}$$

$$P \frac{(V_2)^{\gamma}}{T_2} = \text{Const}$$

$$\text{Slope} = \left( \frac{dP}{dV} \right) = (-\gamma) \left( \frac{P}{V} \right)$$

P



Isotherm

Adiabate

$$\left( \frac{\text{Slope of Adiabate}}{\text{Slope of Isotherm}} \right) = \gamma$$

$$\begin{aligned} W &= \int P dV = \int_{V_1}^{V_2} V^{\gamma} \cdot PV^{\gamma} dV \\ &= \left( \int_{V_1}^{V_2} V^{\gamma} dV \right) (PV_1^{\gamma}) \end{aligned}$$

↑ Const.

$$= (PV_1^{\gamma}) \left( \frac{V_2^{\gamma+1}}{\gamma+1} \right)$$

$$W = (-1) \left( \frac{P_2 V_2 - P_1 V_1}{\gamma-1} \right)$$

 $\Rightarrow$ 

$$W = (-1) \left( \frac{nR}{\gamma-1} (T_2 - T_1) \right)$$

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$$\Rightarrow \text{Const.} = \ln(V) \left( \frac{C_{p,m}}{C_{v,m}} \right) + \ln(P)$$

$$\Rightarrow \text{Const.} = \gamma \ln(V) + \ln(P)$$

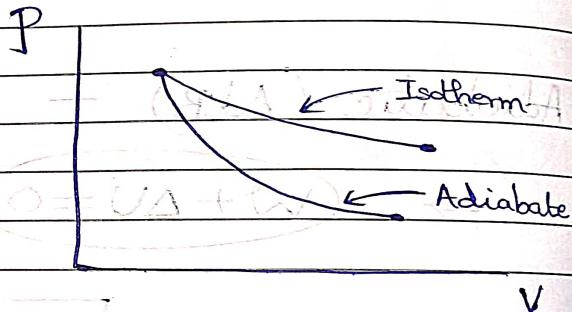
$$PV^\gamma = \text{Const.}$$

Now,

$$TV^{(\gamma=1)} = \text{Const.}$$

$$P^{\frac{(1-\gamma)}{\gamma}} T^{\frac{\gamma}{\gamma-1}} = \text{Const.}$$

$$\text{Slope} = \left( \frac{dP}{dV} \right) = (-\gamma) \left( \frac{P}{V} \right)$$



$$\frac{(\text{Slope of Adiabate})}{(\text{Slope of Isotherm})} = \gamma$$

$$W = \int P dV = \int V \cdot PV^\gamma dV$$

$$= \left( \frac{V_2}{V_1} \right) \left( \frac{P_1 V_1^\gamma}{P_2 V_2^\gamma} \right) = \text{const.}$$

$$= (PV^\gamma) \left( \frac{V_1}{V_2} \right)^{\frac{1}{1-\gamma}}$$

$$W = (-1) \left( \frac{P_2 V_2 - P_1 V_1}{\gamma - 1} \right)$$

$$\Rightarrow W = (-1) (nR) (T_2 - T_1)$$

Q) 2 mol ideal gas obeys  $VT^{1/2} = \text{Const}$ .  
 find work done to inc. its temp by 300K

A)  $VT^{1/2} = \text{Const.} \Rightarrow PV^3 = \text{Const.} \Rightarrow \gamma = 3$

$$(-W) = \left(\frac{nR}{\gamma-1}\right)(T_2 - T_1) = 2 \cdot \frac{25}{3} \cdot 300$$

$$\Rightarrow W = (-2500) \text{ J}$$

Q) Isobaric (iP)  $\rightarrow P = \text{Const.}$

$$q = nC_{p,m}(\Delta T) \rightarrow W = P(\Delta V)$$

$$\Rightarrow W = nR(\Delta T)$$

Q) 14 J heat supplied to diatomic gas. at const. P. find work done.

$$A) \begin{cases} q = nC_{p,m}(\Delta T) \\ W = nR(\Delta T) \end{cases} \Rightarrow W = \left(\frac{R}{C_{p,m}}\right)q_r = \left(\frac{\gamma-1}{\gamma}\right)q$$

$$\Rightarrow W = 4 \text{ J}$$

4) Isochoric ( $iV$ )  $V = \text{Const.}$

$$\Rightarrow W = 0$$

$$\Rightarrow q = \Delta U$$

$$V = \text{Const.}$$

$$(T_1 V_1) - (T_2 V_2) = (T_1 - T_2) V_1 = \Delta TV$$

P

$$\frac{P_1}{P_2} = \frac{(T_1 - T_2) V_1}{(T_1 - T_2) V_2} = \frac{V_1}{V_2}$$

V

(See Pg 149)

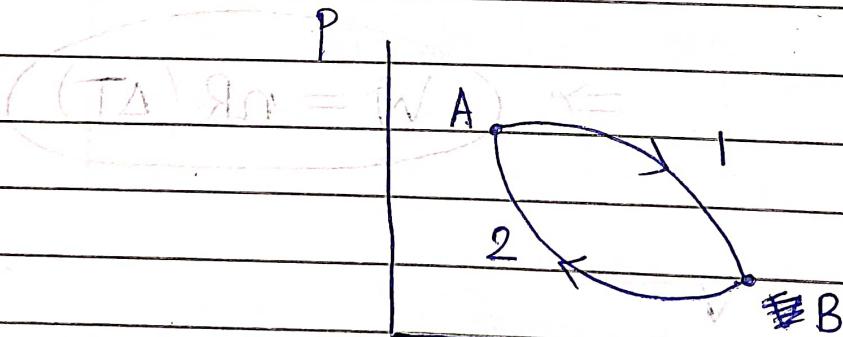
5) Polytropic -  $PV^x = \text{Const.}$

$$P_1 V_1^x = P_2 V_2^x$$

$$C = C_m + R$$

$$= \frac{R}{(1-x)}$$

Cyclic Process



To determine the work done in a cyclic process, we draw the graph of  $P$  vs  $V$  and  $T$  vs  $V$ .

$$W = +(\text{Area of loop})$$

$\circlearrowleft$  if  $\Omega_B$  use  $-$  if  $\Omega$

$$|T \cdot \Delta V| = W$$

Q) find work done.

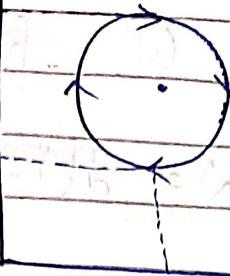
P

2P

P

A)

$$W = +PV \cdot \pi$$



Q) For diatomic gas,

find  $q_{(x-1)}$

P

(2P, 2V)

(P, V)

A)

~~$PV^{\gamma} = \text{Const.} \Rightarrow \gamma = (-1)$~~

~~$W = (-1) \left( \frac{P_2 V_2 - P_1 V_1}{\gamma - 1} \right) = \frac{1}{2} (3PV)$~~

~~$\Delta U = n C_{v,m} (\Delta T) = \frac{1}{2} (\gamma - 1) (P_2 V_2 - P_1 V_1) =$~~

$$A) W = \int P dV = \int_V^{2V} V dV = \left( \frac{V^2}{2} \right)_V^{2V} = \frac{3V^2}{2} = 0 \quad (A)$$

$$\Delta U = n C_{v,m} (\Delta T) = \left( \frac{1}{\gamma - 1} \right) (P_2 V_2 - P_1 V_1) = \left( \frac{5}{2} \right) (3PV) = \frac{15V^2}{2}$$

Now,  $\Delta U = q + (-W) \Rightarrow$

$$q = 9PV$$

★ Q) In above Q, find C.

A)  $dQ - dW = dU \Rightarrow nC_v dT + PdV = nC_{v,m} dT$

If  $PV^x = \text{Const.} \Rightarrow C = C_{v,m} + \left(\frac{P}{n}\right)\left(\frac{dV}{dT}\right)$

$\downarrow$   
 $TV^{(x-1)} = \text{Const.} \Rightarrow C = C_{v,m} + \left(\frac{RT}{V}\right)\left(\frac{V}{T(1-x)}\right)$

$$\left( \Rightarrow V^{(x-1)} + (x-1)V^{(x-2)} T \left( \frac{dV}{dT} \right) = 0 \right)$$

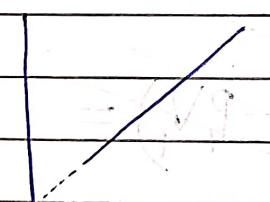
$$\left( \Rightarrow \left( \frac{dV}{dT} \right) = \frac{V}{(1-x)T} \right)$$

$$\Rightarrow C = C_{v,m} + \frac{R}{(1-x)}$$

In this case, diatomic gas &  $x = (-1)$

$$\Rightarrow C = 3R$$

Q)



Diatomc gas.

Find C.

P

A)

$$C = C_{v,m} + \frac{R}{(1-x)} = \frac{5}{2}R + R \Rightarrow C = \frac{7}{2}R$$

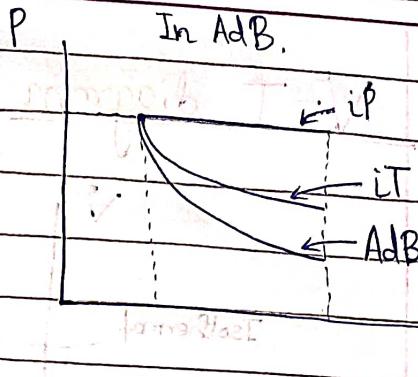
$$(U/\beta) = \text{Const.} \Rightarrow TV = \text{Const.} \Rightarrow PV^2 = \text{Const.} \Rightarrow x = 2$$

$$x = 2$$

$$(V/T = \beta) \Leftrightarrow (V-1) + \beta = C_1$$

- Q) A gas expands from vol.  $V$  to  $2V$  by iP, iT it AdB processes.

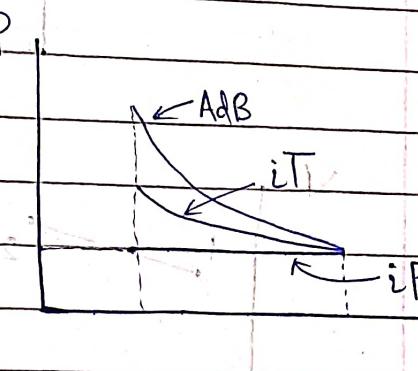
A)



Which has min. P at end?

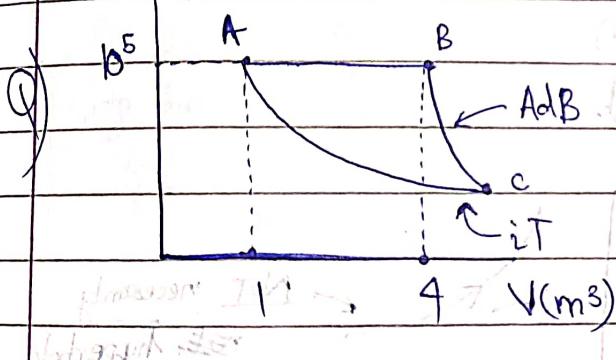
- Q) A gas is compressed from vol.  $2V$  to  $V$  by iP, iT it AdB processes.

A)



Where is  $|W|$  max?

In AdB.

 $P(N/m^2)$ 

A) iT:  $P_A V_A = P_c V_c \Rightarrow P_c V_c = 10^5$

$$P_c = 10^5 / 64$$

AdB:  $P_B V_B^\gamma = P_c V_c^\gamma \Rightarrow P_c V_c^{3/2} = 8 \cdot 10^5$

$$V_c = 64$$

$$W = W_{AB} + W_{BC} + W_{CA} = (10^5)(3) + (-1) \left( \frac{10^5 - 4 \cdot 10^5}{0.5} \right) + (10^5) \ln \left( \frac{1}{4} \right)$$

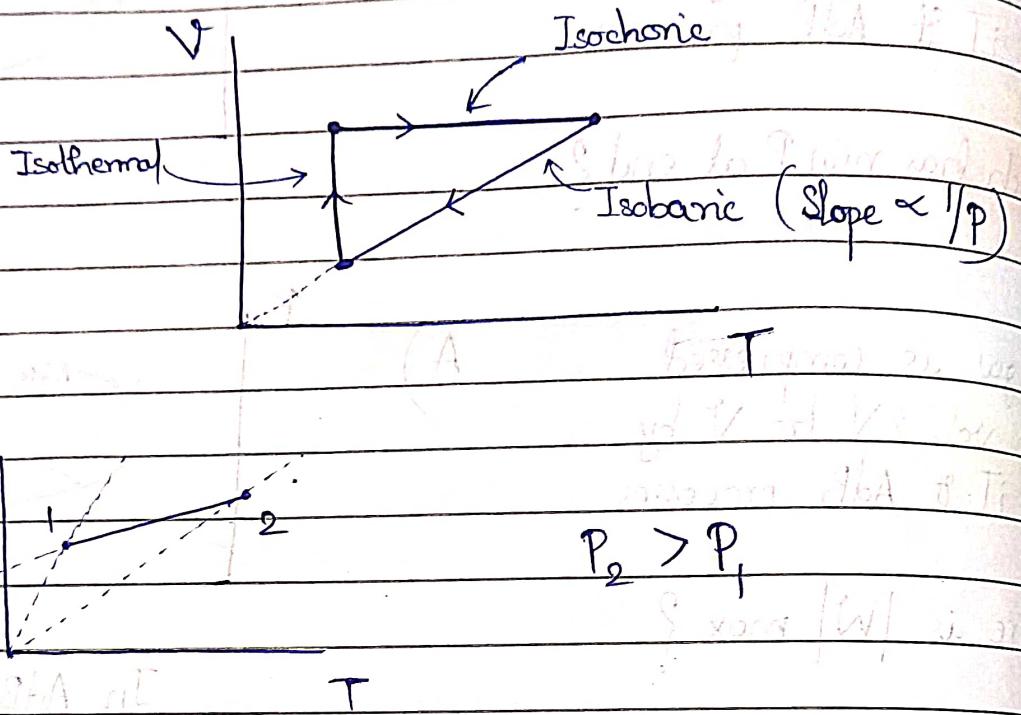
$$= (3 \cdot 10^5 + 6 \cdot 10^5 - \ln(2) \cdot 10^5) = (9 - \ln(2)) \cdot 10^5$$

 $\Rightarrow$ 

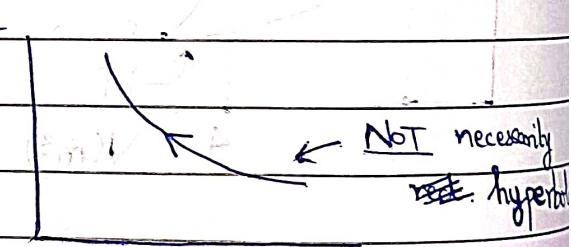
$$W \approx (8.3 \times 10^5) J$$

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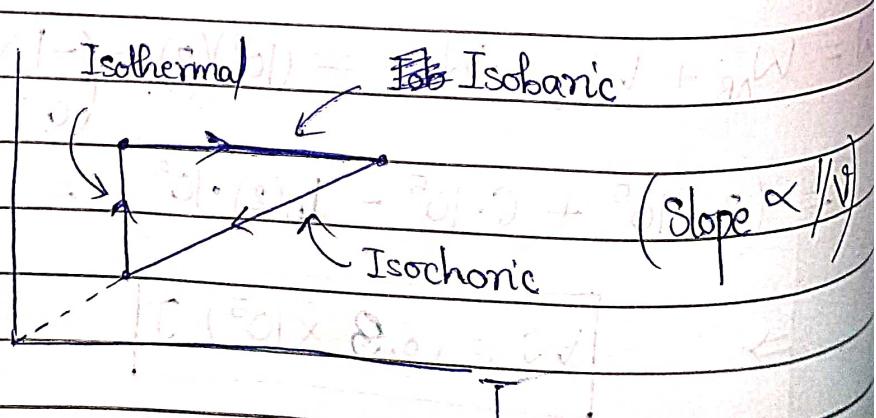
### V-T diagram

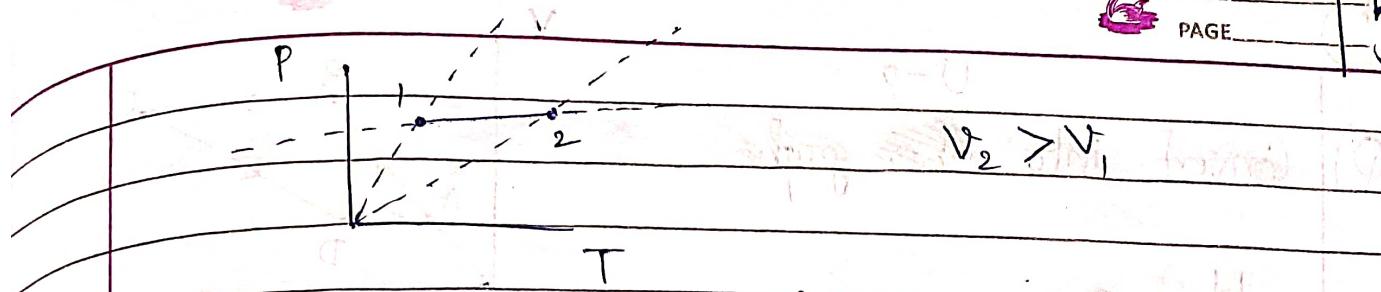


$$\text{for } A \Delta B, \frac{\partial T}{\partial V} = \text{Const.} \quad (\geq 0)$$



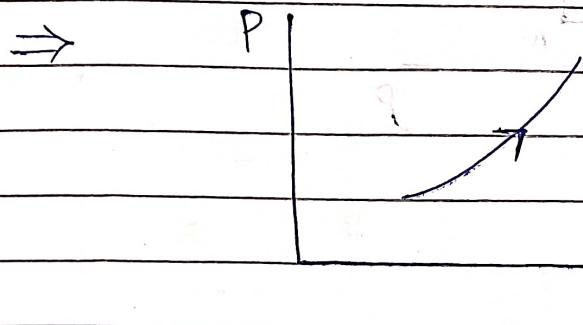
### P-T diagram





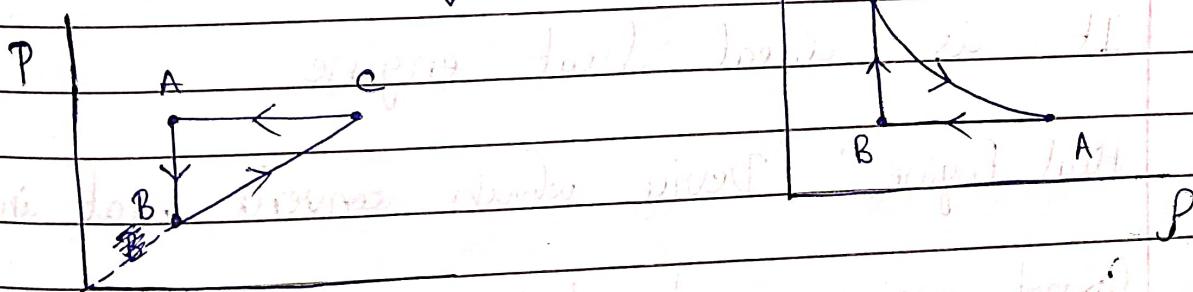
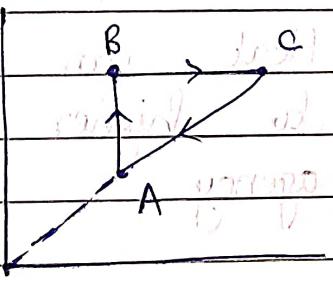
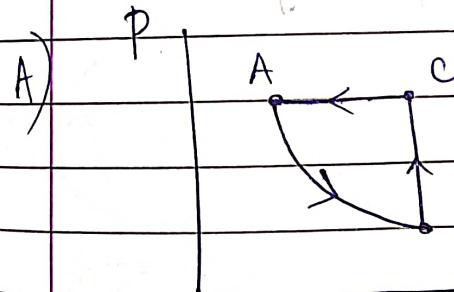
for AdiB,

$$TP \left(\frac{1}{r}\right) = \text{Const.}$$



convert to  $P-V$  diagram

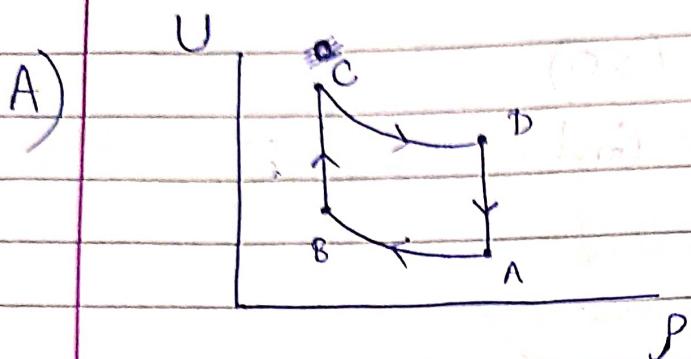
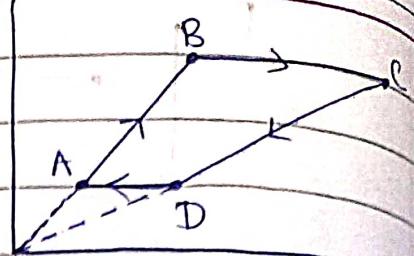
Q) Convert to P-V diagram,  
U-T diagram & P-T diagram



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Q) Convert into ~~v-p~~ graph.



2nd Law of Therm.

Heat can NOT flow from lower temp. to higher temp. without help of external agency.

Carnot Engine

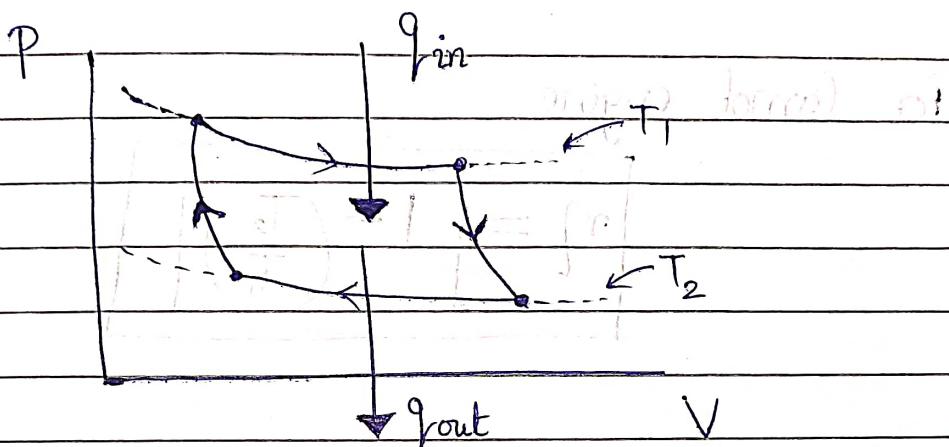
It is ideal heat engine.

Heat Engine: Device which converts heat into work.

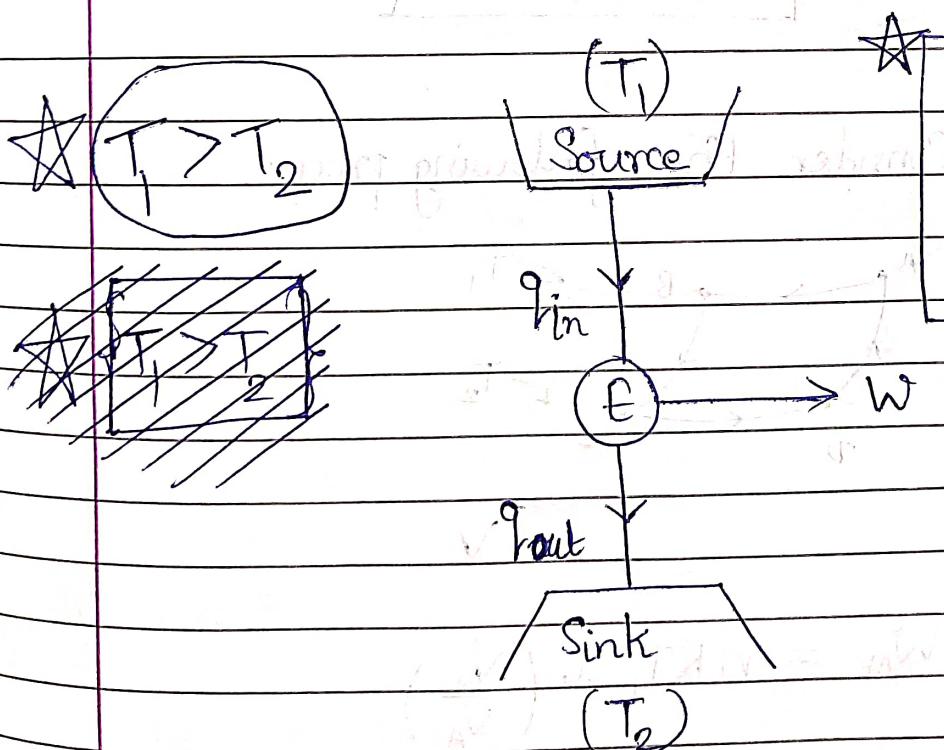
Carnot engine is based on Carnot Cycle.

Carnot Cycle :- A cyclic process consisting of -

- 1) Isothermal Exp<sup>n</sup>
- 3) Isothermal Comp<sup>n</sup>
- 2) Adiabatic Exp<sup>n</sup>
- 4) Adiabatic Comp<sup>n</sup>



During iT exp<sup>n</sup>,  $q_{in}$ ; during iT comp<sup>n</sup>,  $q_{out}$



Since time to complete  
VERY large (as processes  
are reversible)  $\Rightarrow P = 0$   
 $\Rightarrow$  Power of Carnot Engine is Zero.

$q_{in}$  &  $q_{out}$   
are MAGNITUDE  
of heat entering  
it exiting resp.

$$q_{in} = q_{out} + W$$

For any engine,

$$(\text{Efficiency}) = \star \frac{(W)}{q_{\text{in}}}$$

If  $n_{\text{engine}} > 1$

$\Rightarrow$  It doesn't exist!

$$\eta = [1 - \left( \frac{q_{\text{out}}}{q_{\text{in}}} \right)]$$

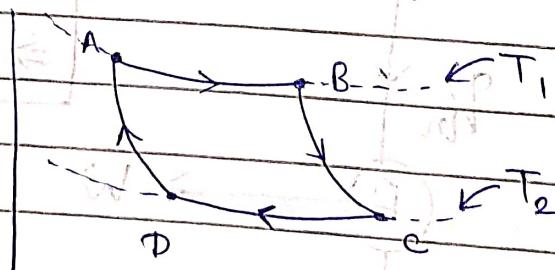
for Carnot engine,

$$\eta = 1 - \left( \frac{T_2}{T_1} \right)$$

Combining gives,

$$\left( \frac{q_{\text{in}}}{T_1} \right) = \left( \frac{q_{\text{out}}}{T_2} \right)$$

Derivation: Consider the following process.



$$q_{\text{in}} = q_{AB} = W_{AB} = nRT_1 \ln \left( \frac{V_B}{V_A} \right)$$

$$q_{\text{out}} = (-q_{CD}) = (-W_{CD}) = (-nRT_2) \ln \left( \frac{V_D}{V_C} \right)$$

(as  $q_{CD} < 0$ )

By AdB,

$$T_1 V_B^{(r-1)} = T_2 V_c^{(r-1)}$$

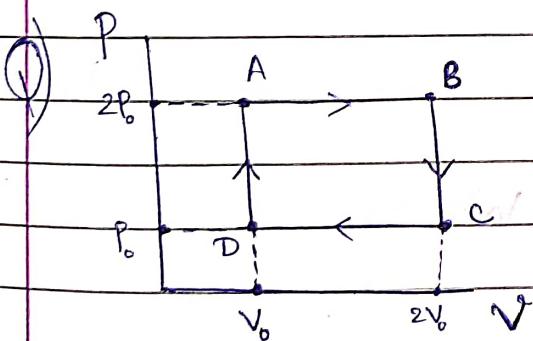
$$st \quad T_1 V_A^{(r-1)} = T_2 V_D^{(r-1)}$$

$$\Rightarrow \left( \frac{V_B}{V_A} \right) = \left( \frac{V_c}{V_D} \right)$$

$$\text{Now, } \left( \frac{q_{in}}{q_{out}} \right) = \frac{nRT_1}{(-nRT_2)} \frac{\ln(V_B/V_A)}{\ln(V_D/V_C)} = \left( \frac{T_1}{T_2} \right)$$

$$\Rightarrow \left[ 1 - \left( \frac{q_{out}}{q_{in}} \right) \right] = \left[ 1 - \left( \frac{T_2}{T_1} \right) \right]$$

$$\Rightarrow \eta = \left[ 1 - \left( \frac{T_2}{T_1} \right) \right]$$



Diatomic gas.

Find efficiency.

$$A) W_{AB} = 2P_0 V_0, \quad W_{BC} = 0, \quad W_{CD} = (-P_0 V_0), \quad W_{DA} = 0$$

$$q_{AB} = n \left( \frac{7}{2} R \right) (\Delta T), \quad q_{BC} = (-ve), \quad q_{DC} = (-ve), \quad q_{DA} = (\Delta U)_{DA}$$

$$= 7P_0 V_0$$

$$q_{out} = q_{BC} + q_{DC} = n \left( \frac{5}{2} R \right) (\Delta T)$$

$$= 5P_0 V_0$$

$$\Rightarrow W = P_0 V_0, \quad q_{in} = \frac{19P_0 V_0}{2} \Rightarrow \eta = \frac{2}{19}$$

Q) Carnot engine. Efficiency =  $1/6$ . If temp. of sink reduced by  $50^\circ\text{C}$ , efficiency gets doubled. Find init. temp. of source & sink.

A)

$$\frac{T_{\text{source}}}{T_{\text{sink}}} \left(1 - \frac{T_2}{T_1}\right) = \frac{1}{6} \Rightarrow \left(\frac{T_2}{T_1}\right) = \frac{5}{6}$$

$$\left(1 - \frac{(T_2 - 50)}{T_1}\right) = \frac{1}{3} \Rightarrow \frac{(T_2 - 50)}{T_1} = \frac{2}{3}$$



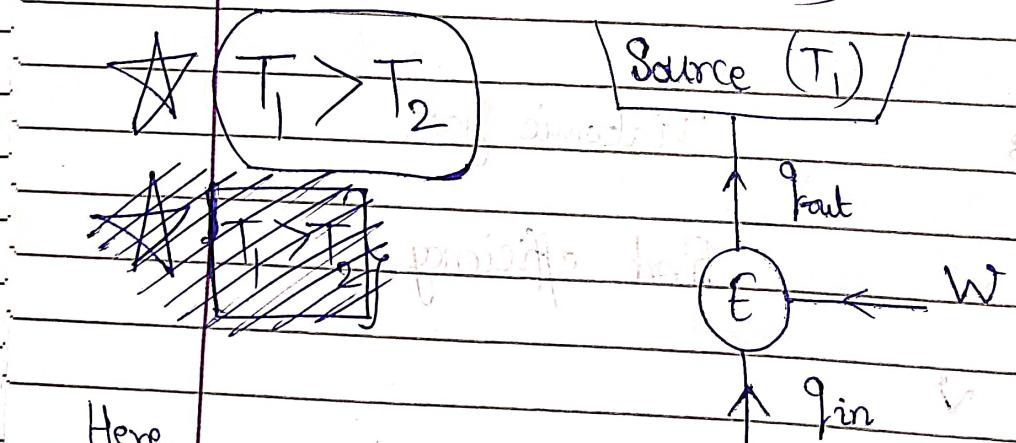
$$T_2 = 250$$

$$T_1 = 300$$

Sink

Source

## Refrigerator (Heat Pump)



Here,

Source = Room

Sink = Fridge

Sink ( $T_2$ )

$$q_{\text{out}} = q_{\text{in}} + W$$



for any heat pump,

$$\text{(Coef. of Performance)} = \left( \frac{q_{in}}{W} \right)$$

$\Rightarrow$

$$\beta = \left( \frac{q_{in}}{q_{out} - q_{in}} \right)$$

for Carnot pump (Carnot cycle ulti chala do),

$$\beta = \left( \frac{T_2}{T_1 - T_2} \right)$$

- Q) Temp. of freezer = ~~(20)~~<sup>(-23)</sup> °C  
Room temp = 27 °C

10 g water at 0 °C  $\rightarrow$  10 g ice at 0 °C

find work done by compressor

A)  $\beta = \left( \frac{q_{in}}{W} \right) = \left( \frac{T_2}{T_1 - T_2} \right) \Rightarrow \frac{(10)(80) \text{ cal}}{W} = \frac{250}{(300 - 250)} = 5$

$$\Rightarrow W = 160 \text{ cal}$$

Carnot Theorem: No engine is more efficient than Carnot engine.

Q) find efficiency of a ~~non~~-Carnot engine

$$T_1 = 500 \text{ K}, \quad T_2 = 300 \text{ K}$$

A)  $\eta_{\text{Carnot}} = \left[ 1 - \left( \frac{T_2}{T_1} \right) \right] = \left( 1 - \frac{3}{5} \right) = 0.4$

$$\Rightarrow \boxed{\eta_{\text{(Non-Carnot Engine)}} < 0.4}$$

$2^{\circ}\text{C}$  = equal to heat  
 $2^{\circ}\text{F}$  = equal to heat

$30^{\circ}\text{F}$  is  $0^{\circ}\text{C}$   $\leftarrow$   $30^{\circ}\text{F}$  is below

more heat than

$$\eta = \frac{Q_1 - Q_2}{Q_1} = \frac{(T_1 - T_2)}{(T_1)} = \frac{(500 - 300)}{500} = 0.4$$

$$100 - 60 = 40$$