

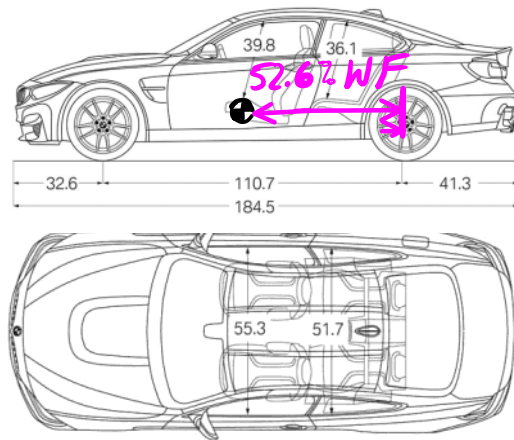
## TANGENT SPEED, CHARACTERISTIC SPEED, AND CRITICAL SPEED

### • BMW M4 VEHICLE PARAMETERS:

- $m = 1630 \text{ [kg]}$
- $l = 2.81 \text{ [m]}$
- $WF = 52.6\% \rightarrow l_F = 1.33 \text{ [m]}, l_R = 1.48 \text{ [m]}$

ESTIMATED FROM SHD PAD:

- $C_{\alpha_F} = 84,316 \text{ [N/rad]}, C_{\alpha_R} = 91,177 \text{ [N/rad]}$
- $\kappa_G = 0.0017 \frac{\text{[rad]}}{\text{[m/s}^2]} \rightarrow \kappa_G > 0 \rightarrow \text{UNDERSTEER}$



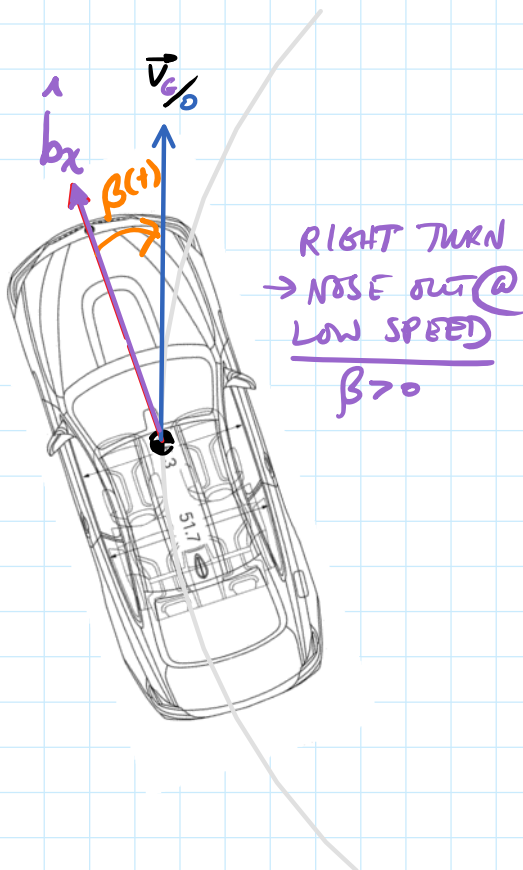
## TANGENT SPEED, $V_T$

- THE TANGENT SPEED,  $V_T$ , IS THE SPEED AT WHICH THE NOSE OF THE VEHICLE IS ALIGNED WITH THE VELOCITY VECTOR ( $\hat{b}_x$  IS ALIGNED WITH  $\vec{v}_{c/d}$ ) WHILE CORNERING A CONSTANT RADIUS CORNER, AT CONSTANT SPEED

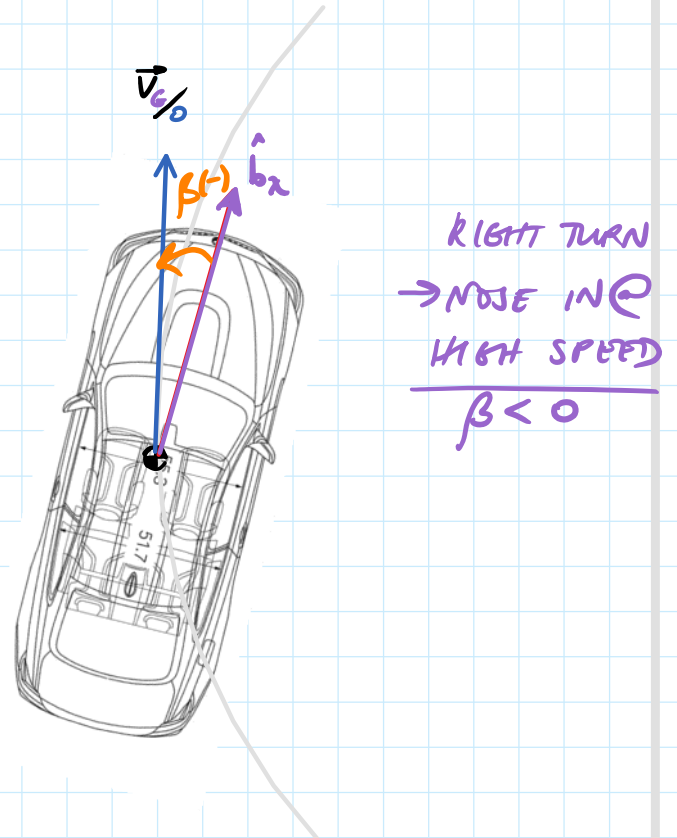
## CORNER, AT CONSTANT SPEED

- IN OTHER WORDS, THIS IS THE SPEED AT WHICH A VEHICLE CORNERS WITH NO STEADY-STATE SIDESLIP.
- BELOW THE TANGENT SPEED, A VEHICLE WILL HAVE A "NOSE-OUT" ORIENTATION. ABOVE THE TANGENT SPEED, A VEHICLE WILL HAVE A NOSE-IN ORIENTATION.

BELOW TANGENT SPEED:



ABOVE TANGENT SPEED:



- INTERESTINGLY, THE TANGENT SPEED IS INDEPENDENT OF THE STEER ANGLE (FOR SMALL STEER ANGLES). THIS MEANS, REGARDLESS OF THE RADIUS YOU ARE CORNERING, THE TANGENT SPEED WILL BE THE SAME.
- WE CAN DERIVE THE TANGENT SPEED BY EXAMINING THE CONDITIONS FOR WHICH THE STEADY-STATE SIDESLIP ANGLE IS ZERO!

$$\lim_{t \rightarrow \infty} \beta(t) = \lim_{s \rightarrow 0} s \beta(s) = \beta^* = G_{\beta}^* \delta^*$$

- $\beta^*$  IS ZERO IF  $G_{\beta}^* = 0$ . THEREFORE, AT THE TANGENT SPEED

$$G_{\beta}^* = \frac{N_{\delta} (Y_r - m V_T) - Y_{\delta} N_r}{N_r Y_{\beta} - N_{\beta} Y_r + N_{\beta} m V_T} = 0$$

- THEREFORE, THE NUMERATOR MUST BE ZERO

$$N_{\delta} (Y_r - m V_T) - Y_{\delta} N_r = 0$$

- ALSO, REMEMBER  $Y_r$  AND  $N_r$  ARE FUNCTIONS OF THE VEHICLE VELOCITY, SO LET US EXPAND EVERYTHING OUT:

$$C_{dF} l_F \left( \frac{C_{dR} l_R - C_{dF} l_F}{V_T} - m V_T \right) + \frac{C_{dF} (C_{dF} l_F^2 + C_{dR} l_R^2)}{V_T} = 0$$

- MULTIPLYING BY  $V_T$ , AND DIVIDING BY  $C_{dF}$

$$\cancel{C_{dF} C_{dR} l_F l_R} - \cancel{C_{dF}^2 l_F^2} - m V_T^2 \cancel{C_{dF} l_F} + \cancel{C_{dF}^2 l_F^2} + \cancel{C_{dF} C_{dR} l_R^2} = 0$$

$$\rightarrow C_{dR} l_R (\underbrace{l_F + l_R}_l) - m V_T^2 l_F = 0$$

$$\rightarrow V_T = \sqrt{\frac{l}{m} \cdot \frac{l_R}{l_F} \cdot C_{dR}}$$

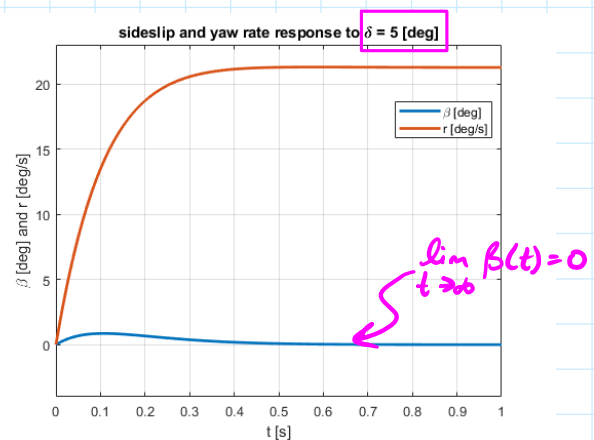
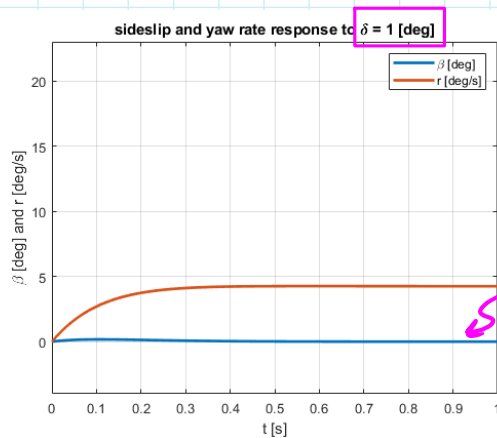
EXAMPLE 19-1: TANGENT SPEED BMW M4

$$V_T = \sqrt{\frac{2.81 [m]}{1630 [kg]} \cdot \frac{1.48 [m]}{1.33 [m]} \cdot 91,177 \frac{[rad]}{[m/s^2]}}$$

$$V_T = \sqrt{1630[\text{kg}] \cdot 1.33[\text{m}] \cdot 71.1 \text{ rad/s}} [\text{m/s}^2]$$

$$= 13.21 [\text{m/s}] = 29.5 [\text{mph}]$$

- LOOKING AT STEP RESPONSE AT TANGENT SPEED, COMPARING  $\delta = 1 [\text{deg}]$  AND  $\delta = 5 [\text{deg}]$



- IN BOTH CASES, THE SIDESLIP IS ZERO, EVEN THOUGH THE YAW RATE AND LATERAL ACCELERATION ARE VERY DIFFERENT!

CHARACTERISTIC SPEED,  $V_{\text{CHAR}}$

- THE CHARACTERISTIC SPEED IS THE SPEED AT WHICH AN UNDERSTEER CAR PRODUCES

WHICH AN **UNDERSTEER** CAR PRODUCES  
THE MAXIMUM YAW RATE PER STEER ANGLE

- THE CHARACTERISTIC SPEED CAN ALSO BE A MEASURE OF THE AMOUNT OF UNDERSTEER A CAR HAS: THE LOWER THE CHARACTERISTIC SPEED, THE MORE UNDERSTEER A CAR IS!

~ THE CHARACTERISTIC SPEED OF MOST PASSENGER CARS IS AROUND 40-50 [mph]

- LET US DERIVE A GENERAL EXPRESSION FOR THE CHARACTERISTIC SPEED OF AN UNDERSTEER VEHICLE.

$$\lim_{t \rightarrow \infty} r(t) = \lim_{s \rightarrow 0} sr(s) = r^* = G_r^* \delta^*$$

- $V_{CHAR}$  IS THE SPEED AT WHICH  $G_r^* = \frac{r^*}{\delta^*}$  IS MAXIMIZED

$$G_r^* = \frac{N_\beta \gamma_\delta - N_\delta \gamma_\beta}{N_r \gamma_\beta - N_\beta \gamma_r + N_\beta mV}$$

$$= \frac{C_{dR} C_{dF} l}{C_{dR} C_{dF} l^2} \cdot \frac{\frac{V}{C_{dR} C_{dF} l^2}}{\frac{V}{C_{dR} C_{dF} l^2}}$$

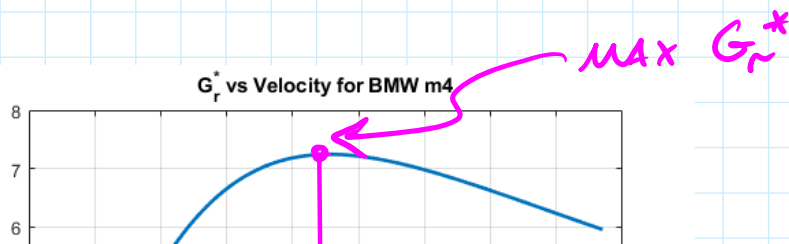
$$\begin{aligned}
 &= \frac{\frac{C_{aF} C_{aR} l^2}{V} + (C_{aR} l_R - C_{aF} l_F) m V}{\frac{V}{l}} \cdot \frac{V}{C_{aF} C_{aR} l^2} \\
 &= \frac{\frac{V}{l}}{1 + \frac{(C_{aR} l_R - C_{aF} l_F) m V}{C_{aF} C_{aR} l^2}} \\
 &= \frac{\frac{V}{l}}{1 + \boxed{\frac{uG}{l}} \cdot V^2}
 \end{aligned}$$

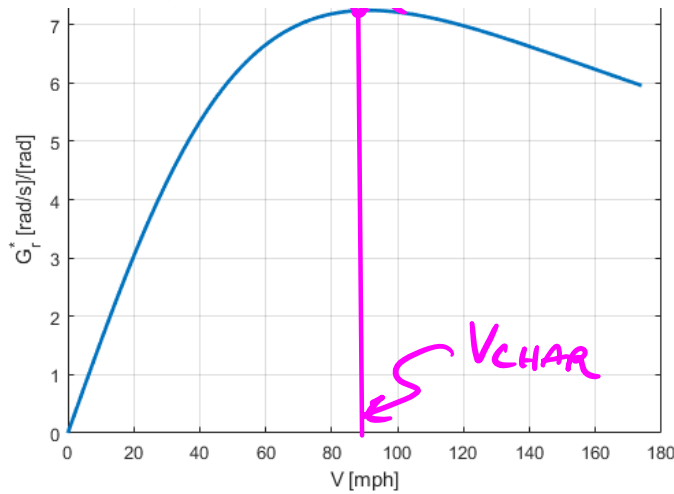
DEFINED AS THE "STABILITY FACTOR",  $K$

$$K \triangleq \frac{uG}{l}$$

$$\rightarrow G_r^* = \frac{\frac{V}{l}}{1 + KV^2}$$

• LET US PLOT  $G_r^*$  FOR THE m4:





- BY VISUAL INSPECTION, WE CAN IDENTIFY THE CHARACTERISTIC SPEED SOMEWHERE AROUND 90 [mph], BUT IT IS MORE INSTRUCTIVE TO DERIVE AN ANALYTICAL EXPRESSION FOR  $V_{CHAR}$ .

- $G_r^*$  IS MAXIMIZED WHEN  $\frac{\partial G_r^*}{\partial V} = 0$

$$\frac{\partial G_r^*}{\partial V} = \frac{\partial}{\partial V} \left( \frac{\frac{V}{\ell}}{1 + kV^2} \right)$$

$$= \frac{1}{\ell} (1 + kV^2)^{-1} - \frac{V}{\ell} (1 + kV^2)^{-2} 2kV$$

$$= \frac{1 - kV^2}{\ell (1 + kV^2)^2}$$



$$\left. \frac{\partial G_r^*}{\partial V} \right|_{V=V_{\text{CHAR}}} = \frac{1 - KV_{\text{CHAR}}^2}{l(1 + KV_{\text{CHAR}}^2)^2} = 0$$

$$\rightarrow 1 - KV_{\text{CHAR}}^2 = 0$$

$$\rightarrow V_{\text{CHAR}} = \sqrt{\frac{1}{K}} = \sqrt{\frac{l}{UG}}$$

- **NOTICE:**  $V_{\text{CHAR}}$  INCREASES AS  $UG$  GETS SMALLER
- A NEUTRAL STEER CAR HAS  $V_{\text{CHAR}} \rightarrow \infty$
- $V_{\text{CHAR}}$  DOES NOT EXIST FOR OVERSTEER CARS! (BECAUSE  $UG < 0$ )

**EXAMPLE 19-2:**  $V_{\text{CHAR}}$  FOR BMW M4

- THE UNDERSTEER GRADIENT IS

$$UG = \frac{m}{l} \frac{C_{aR}l_R - C_{aF}l_F}{C_{aR}C_{aF}} = 0.0017 \frac{[\text{rad}]}{[\text{m/s}^2]}$$

• THE CRITICAL SPEED IS

- THE STABILITY FACTOR IS:

$$k = \frac{uG}{l} = \frac{0.0017 \frac{[\text{rad}]}{[\text{m/s}^2]}}{2.81 [\text{m}]} = 6.0262 \times 10^{-4} \frac{[\text{rad}]}{[\text{m/s}^2]}$$

- THE CHARACTERISTIC VELOCITY IS:

$$V_{\text{CHAR}} = \sqrt{\frac{1}{k}} = 40.74 [\text{m/s}] = 91.1 [\text{mph}]$$

↓ CRITICAL SPEED,  $V_{\text{CRIT}}$  ↓

- THE CRITICAL SPEED,  $V_{\text{CRIT}}$ , ONLY APPLIES TO OVERSTEER CARS ( $uG < 0$ )
- THE CRITICAL SPEED IS THE SPEED AT WHICH AN OVERSTEER CAR HAS A THEORETICALLY INFINITE YAW RATE IN RESPONSE TO ANY STEERING INPUT (THE SLIGHTEST STEERING DISTURBANCE WILL SPIN OUT THE CAR!)

- MATHEMATICALLY, THIS MEANS

$$G_r^* \bigg|_{V_{\text{CRIT}}} = \frac{\frac{V_{\text{CRIT}}}{l}}{1 + K \cdot V_{\text{CRIT}}^2} = \infty$$

- THIS GIVES US THE CONDITION

$$1 + K \cdot V_{\text{CRIT}}^2 = 0$$

$$\rightarrow V_{\text{CRIT}} = \sqrt{-\frac{1}{K}} = \sqrt{-\frac{l}{uG}}$$

THIS GIVES A REAL RESULT  
ONLY IF  $uG < 0$

### EXAMPLE 19-3: MAKING THE m4 OVERSTEER

- WE COULD TRY TO MAKE THE BMW m4 OVERSTEER BY SIGNIFICANTLY INCREASING THE REAR TIRE PRESSURE TO DECREASE THE REAR CORNERING STIFFNESS.
- IN ORDER TO MAKE THE CAR OVERSTEER,

WE NEED

$$N_B = C_{\alpha_R} l_R - C_{\alpha_F} l_F < 0$$

$$\rightarrow C_{\alpha_R} l_R < C_{\alpha_F} l_F$$

$$\rightarrow C_{\alpha_R} < \frac{l_F}{l_R} C_{\alpha_F} = \frac{1.3328 \text{ [m]}}{1.479 \text{ [m]}} 84316 \left[ \frac{\text{N}}{\text{rad}} \right]$$

$$\rightarrow C_{\alpha_R} < 75,981 \left[ \frac{\text{N}}{\text{rad}} \right]$$

- LET US SAY WE REDUCE  $C_{\alpha_R}$  TO 70,000  $\left[ \frac{\text{N}}{\text{rad}} \right]$ .
- THE NEW UNDERSTEER GRADIENT IS

$$\begin{aligned} UG &= \frac{m}{l} \frac{C_{\alpha_R} l_R - C_{\alpha_F} l_F}{C_{\alpha_R} C_{\alpha_F}} \\ &= -8.686 \times 10^{-4} \frac{[\text{rad}]}{\left[ \frac{\text{m}}{\text{s}^2} \right]} \end{aligned}$$

- THIS GIVES US

$$V_c = \sqrt{-\frac{l}{UG}} = 56.90 \text{ [m/s]} = 127.27 \text{ [mph]}$$

$$V_{crit} = \sqrt{-\frac{l}{uG}} = 56.90 \text{ [m/s]} = 127.27 \text{ [mph]}$$

- IF WE PLOT  $G_r^* = \frac{\ddot{r}^*}{\delta^*}$  VS.  $V$  FOR THIS OVERSTEER CONFIGURATION, WE EASILY SEE THE INSTABILITY

