

Turbine

state 1

$$T_1 = 400^\circ\text{C} = 673\text{K} \quad T_2 = 100^\circ\text{C} = 373\text{K}$$

$$P_1 = ?$$

$$P_2 = 200\text{kPa}$$

$$V_1 = 20\text{m/s}$$

$$V_2 = 5\text{m/s}$$

$$A_1 = 2\text{m}^2$$

$$\begin{aligned} E_{out} &= E_{in} \\ (E_{out} + KE_{out} + H_{out}) &= KE_{in} + H_{in} \\ -W_{out} &= \Delta H + \Delta KE \end{aligned}$$

$$-W_{out} = \dot{m}(\Delta h + \Delta ke)$$

$$\therefore 300 \leq \Delta T \leq 500 \rightarrow \Delta h = C_p(T_2 - T_1) \quad (\text{Table A2b})$$

$$\frac{C_p - 1.029}{523 - 500} = \frac{1.040 - 1.029}{550 - 500}$$

$$C_p = 1.034 \text{ kJ/kg}\cdot\text{K}$$

$$P_{1A} V_{1A} = R_{air} T_{1A}$$

$$\therefore = P_{1A} T / D$$

$$P_{IA} V_{IA} = K_{air} T_{IA}$$

$$V_{IA} = R_{air} T_{IA} / P_{IA}$$

$$\dot{m}_A = \frac{A_{IA} V_{IA}^0}{V_{IA}} = \frac{P_{IA} A_{IA} V_{IA}^0}{R_{air} T_{IA}}$$

$$(\dot{\omega}_{out})_A = - \frac{P_{IA} A_{IA} V_{IA}^0}{R_{air} T_{IA}} \left(C_p (T_{2A} - T_{IA}) + \frac{1}{2} \frac{(V_{2A}^0 - V_{IA}^0)^2}{1000} \right) \quad (1)$$

Compressor

State 1

$$T_{1B} = 150^\circ C$$

$$P_{1B} = 100 kPa$$

$$V_{1B}^0 = 0 m/s$$

$$\dot{m}_B = 10 kg/s$$

State 2

$$T_{2B} = 500^\circ C$$

$$P_{2B} = 2 MPa$$

$$V_{2B}^0 = 100 m/s$$

@ 100 kPa, $T_1 > T_{sat}$ \Rightarrow superheated

$$h_{1B} = 2776.6 \text{ kJ/kg from Table A6}$$

@ 2 MPa, $T_{2B} > T_{sat}$

$$h_{2B} = 3468.3 \text{ kJ/kg from Table A6}$$

$$\dot{E}_{in} = \dot{E}_{out}$$

$$H_{in} + KE_{in} + \dot{W}_{in} = H_{out} + KE_{out}$$

$$(\dot{\omega}_{in})_B = \dot{m}_B [(h_{2B} - h_{1B}) + \frac{1}{2} (V_{2B}^0 - V_{1B}^0)^2]$$

$$(\dot{\omega}_{in})_B = \dot{m}_B \left[(h_{2B} - h_{1B}) + \frac{1}{2} \frac{(V_{2B}^0 - V_{1B}^0)^2}{1000} \right] \quad (2)$$

$$\cancel{(\dot{\omega}_{in})_B = (\dot{\omega}_{out})_A = \dot{\omega}}$$

$$\dot{\omega} = \dot{m}_B \left[(h_{2B} - h_{1B}) + \frac{1}{2} \frac{(V_{2B}^0 - V_{1B}^0)^2}{1000} \right] = - \frac{P_{IA} A_{IA} V_{IA}^0}{R_{air} T_{IA}} \left(C_p (T_{2A} - T_{IA}) + \frac{1}{2} \frac{(V_{2A}^0 - V_{IA}^0)^2}{1000} \right)$$

$$\dot{\omega} = (10) ((3468.3 - 2776.6) + \frac{1}{2} \frac{(100^2 - 0^2)}{1000})$$

a) $\boxed{\dot{\omega} = 6967 \text{ kW}}$

$$\dot{N} = - P_{IA} A_{IA} V_{IA}^0 / (r_{-1T} - r_{+1} + 1/(10^2 \cdot 10^2))$$

$$\ddot{\omega} = -\frac{P_{IA} A_{IA} V_{IA}^0}{R_{air} T_{IA}} \left(C_p (T_{2A} - T_{1A}) + \frac{1}{2} \frac{(V_{2A}^0 - V_{1A}^0)^2}{1000} \right)$$

$$6967 = -\frac{P_{IA}(1)(20)}{(0.287)(400+273)} \left[1.034(100-400) + \frac{1}{2000} (5^2 - 20^2) \right]$$

$$6967 = P_{IA}(32.139)$$

b) $P_{IA} = 216.77 \text{ kPa}$

$$\dot{m}_B = \frac{A_{2B} V_{2B}^0}{V_{2B}}$$

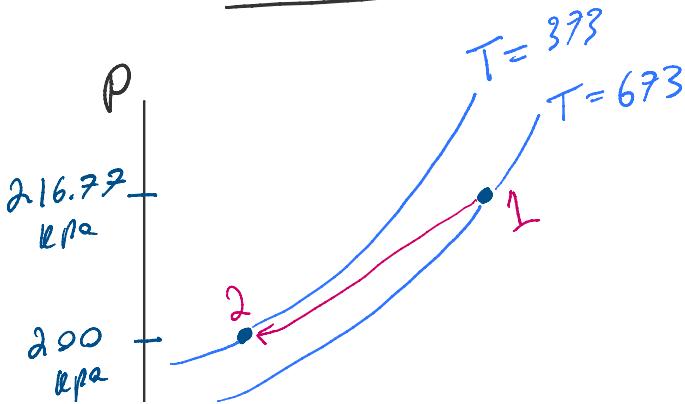
$$A_{2B} = \frac{\dot{m}_B V_{2B}}{V_{2B}^0} = \frac{(10) V_{2B}}{100} = \frac{V_{2B}}{10}$$

@ $P_{2B} = 2 \text{ MPa}, 500^\circ\text{C} > T_{sat} \therefore \text{superheated}$

$$V_{2B} = 0.17568 \text{ m}^3/\text{kg}$$

c) $A_{2B} = 0.17568 / 10 = 0.017568 \text{ m}^2$

d) Turbine



Compressor

