

Mechatronics 3DX4

Slides 7: Steady-State Errors

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Material based on lecture notes by P. Taylor and M. Lawford, and *Control Systems Engineering* by N. Nise.

Introduction

- ▶ We now focus on the third design specification, *steady-state error*.
- ▶ We define **steady-state error** to be the difference between input and output as $t \rightarrow \infty$.
- ▶ We will see that control system design typically means we will have to make trade-offs between the desired transient, steady-state, and stability specifications.

Test Inputs

- ▶ Table below shows the standard test inputs typically used for evaluating steady-state error.




Waveform	Name	Physical interpretation	Time function	Laplace transform
	Step	Constant position	1	$\frac{1}{s}$
	Ramp	Constant velocity	t	$\frac{1}{s^2}$
	Parabola	Constant acceleration	$\frac{1}{2}t^2$	$\frac{1}{s^3}$

Table 7.1.

Choosing a Test Inputs

- ▶ The test inputs we will choose for our steady-state analysis and design depends on our target application.

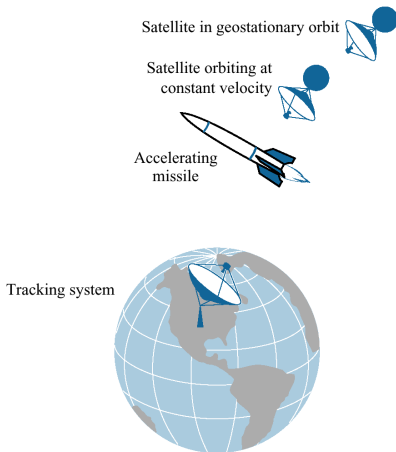


Figure 7.1.

Steady-State Error and Stable Systems

- ▶ The calculations we will be deriving for steady-state apply only to stable systems.
- ▶ Unstable systems represent loss of control in steady-state as the transient response swamps the forced response.
- ▶ As we analyze and design a system for steady-state error, we must constantly check the system for stability.

Steady-State Error and Step Inputs

- ▶ With step inputs, we can get two types of steady-state errors:
 1. Zero error.
 2. A constant error value.

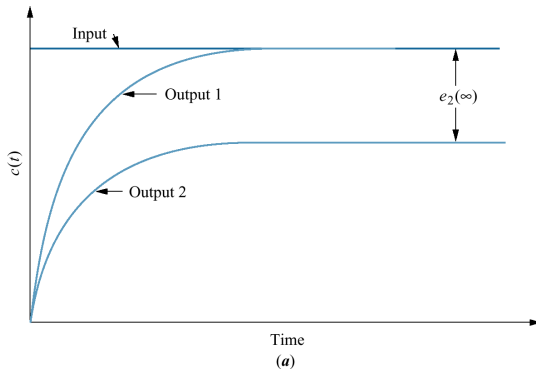


Figure 7.2.

Steady-State Error and Ramp input

- ▶ With ramp inputs, we can get three types of steady-state errors:
 1. Zero error.
 2. A constant error value.
 3. Infinite error.

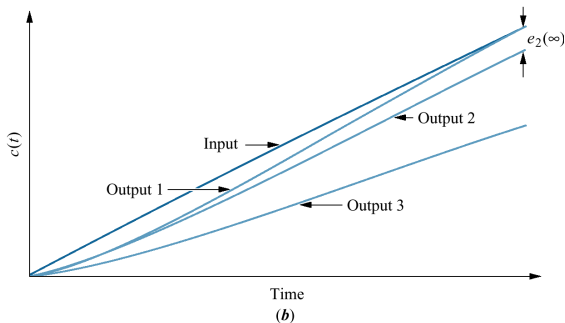


Figure 7.2.

Steady-State Error and Block Diagrams

- ▶ If we have a closed-loop transfer function $T(s)$, we can represent our error signal, $E(s)$, as in figure (a).
- ▶ We are interested in the time domain signal, $e(t) = \mathcal{L}^{-1}\{E(s)\}$, as $t \rightarrow \infty$.
- ▶ If we have a unity feedback system, we already have $E(s)$ as part of our diagram, as shown in figure (b).

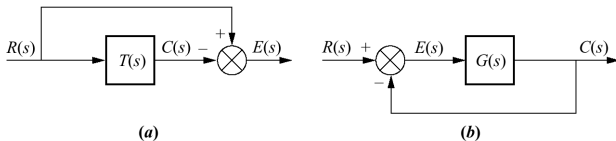


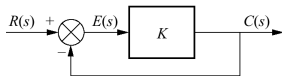
Figure 7.3.

Sources of Steady-State Error

- ▶ Steady-state errors can arise from nonlinear sources, such as backlash in gears or motors requiring a minimum input voltage before it starts to move.
- ▶ Steady-state errors can also arise from configuration of system and the input we apply.
- ▶ Consider a step input applied to the system below which has constant gain.
- ▶ If a unity feedback system has a feedforward transfer function $G(s)$, then we can derive the transfer function $\frac{E(s)}{R(s)}$ as follows:

$$C(s) = E(s)G(s) \quad (1)$$

$$E(s) = R(s) - C(s) \quad (2)$$



(a)

Figure 7.4.

Sources of Steady-State Error - II

- ▶ Substituting equation 1 into equation 2 gives:

$$\begin{aligned}E(s) &= R(s) - E(s)G(s) \\E(s)[1 + G(s)] &= R(s) \\ \frac{E(s)}{R(s)} &= \frac{1}{1 + G(s)}\end{aligned}\tag{3}$$

- ▶ For $G(s) = K$, we get

$$\frac{E(s)}{R(s)} = \frac{1}{1 + K}\tag{4}$$

- ▶ For $R(s) = \frac{1}{s}$ (unit step), we get $E(s) = \frac{1}{s(1+K)}$.

- ▶ We thus have $e_{ss} = \lim_{t \rightarrow \infty} e(t) = \lim_{s \rightarrow 0} sE(s) = \frac{1}{1+K}$

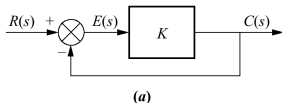


Figure 7.4.

Sources of Steady-State Error - III

$$\frac{E(s)}{R(s)} = \frac{1}{1 + G(s)}$$

- ▶ If we add an integrator to the forward-path gain, we get $G(s) = \frac{K}{s}$ giving

$$\frac{E(s)}{R(s)} = \frac{1}{1 + \frac{K}{s}} = \frac{s}{s + K} \quad (5)$$

- ▶ For $R(s) = \frac{1}{s}$ (unit step), we get $E(s) = \frac{1}{(s+K)}$.
- ▶ We thus have

$$e_{ss} = \lim_{t \rightarrow \infty} e(t) = \lim_{s \rightarrow 0} sE(s) = \frac{0}{0 + K} = 0 \quad (6)$$

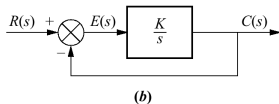


Figure 7.4.

Steady-State Error and $T(s)$

- ▶ In Diagram below, we have $E(s) = R(s) - C(s)$.
- ▶ We also have:

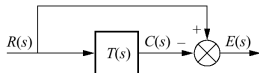
$$C(s) = R(s)T(s) \quad (7)$$

- ▶ Combining the two we get

$$E(s) = R(s) - R(s)T(s) = R(s)[1 - T(s)] \quad (8)$$

- ▶ We thus have

$$\begin{aligned} e_{ss} &= \lim_{s \rightarrow 0} s E(s) \\ &= \lim_{s \rightarrow 0} s R(s)[1 - T(s)] \end{aligned} \quad (9)$$



(a)

Figure 7.3.

Steady-State Error and $G(s)$

- From equation 3, we have

$$E(s) = \frac{R(s)}{1 + G(s)} \quad (10)$$

- We thus have

$$\begin{aligned} e_{ss} &= \lim_{s \rightarrow 0} s E(s) \\ &= \lim_{s \rightarrow 0} s \frac{R(s)}{1 + G(s)} \end{aligned} \quad (11)$$

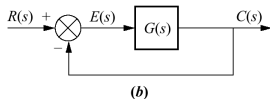


Figure 7.3.

Steady-State Error, $G(s)$, and Step Input

- ▶ For input $R(s) = \frac{1}{s}$, we get

$$e_{ss} = \lim_{s \rightarrow 0} s \frac{1/s}{1 + G(s)} = \frac{1}{1 + \lim_{s \rightarrow 0} G(s)} \quad (12)$$

- ▶ We refer to the term $\lim_{s \rightarrow 0} G(s)$ as **dc gain of the forward transfer function**.
- ▶ To have zero steady-state error we need

$$\lim_{s \rightarrow 0} G(s) = \infty \quad (13)$$

- ▶ For $G(s)$ of form below, we thus need $n \geq 1$

$$G(s) \equiv \frac{(s + z_1)(s + z_2) \cdots}{s^n (s + p_1)(s + p_2) \cdots} \quad (14)$$

- ▶ If $n = 0$, we get

$$\lim_{s \rightarrow 0} G(s) = \frac{(0 + z_1)(0 + z_2) \cdots}{(0 + p_1)(0 + p_2) \cdots} = \frac{z_1 z_2 \cdots}{p_1 p_2 \cdots} \quad (15)$$

Steady-State Error, $G(s)$, and Ramp Input

- ▶ For input $R(s) = \frac{1}{s^2}$, we get

$$e_{ss} = \lim_{s \rightarrow 0} \frac{s(1/s^2)}{1 + G(s)} = \lim_{s \rightarrow 0} \frac{1}{s + sG(s)} = \frac{1}{\lim_{s \rightarrow 0} sG(s)} \quad (16)$$

- ▶ To have zero steady-state error for ramp input, we need

$$\lim_{s \rightarrow 0} sG(s) = \infty \quad (17)$$

- ▶ For $G(s)$ of form below, we thus need $n \geq 2$

$$G(s) \equiv \frac{(s + z_1)(s + z_2) \cdots}{s^n(s + p_1)(s + p_2) \cdots} \quad (18)$$

- ▶ If $n = 1$, we get

$$\lim_{s \rightarrow 0} sG(s) = \frac{z_1 z_2 \cdots}{p_1 p_2 \cdots} \quad (19)$$

- ▶ If $n = 0$, we get

$$\lim_{s \rightarrow 0} sG(s) = \frac{s(s + z_1)(s + z_2) \cdots}{(s + p_1)(s + p_2) \cdots} = 0 \quad (20)$$

Steady-State Error, $G(s)$, and Parabolic Input

- ▶ For input $R(s) = \frac{1}{s^3}$, we get

$$e_{ss} = \lim_{s \rightarrow 0} \frac{s(1/s^3)}{1 + G(s)} = \lim_{s \rightarrow 0} \frac{1}{s^2 + s^2 G(s)} = \frac{1}{\lim_{s \rightarrow 0} s^2 G(s)} \quad (21)$$

- ▶ To have zero steady-state error for a parabolic input, we need

$$\lim_{s \rightarrow 0} s^2 G(s) = \infty \quad (22)$$

- ▶ For $G(s)$ of form below, we thus need $n \geq 3$

$$G(s) \equiv \frac{(s + z_1)(s + z_2) \cdots}{s^n (s + p_1)(s + p_2) \cdots} \quad (23)$$

- ▶ If $n = 2$, we get

$$\lim_{s \rightarrow 0} s^2 G(s) = \frac{z_1 z_2 \cdots}{p_1 p_2 \cdots} \quad (24)$$

- ▶ If $n = 1$, we get

$$\lim_{s \rightarrow 0} s^2 G(s) = \frac{s(s + z_1)(s + z_2) \cdots}{(s + p_1)(s + p_2) \cdots} = 0 \quad (25)$$

Steady-State Error eg.

- Find the steady state errors for inputs $5u(t)$, $5tu(t)$, and $\frac{5}{2}t^2u(t)$.

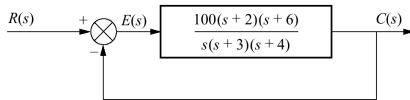


Figure 7.6.

Static Error Constants

- We now define steady state-error performance specifications called **static error constants**.

1. Position Constant: $K_p = \lim_{s \rightarrow 0} G(s)$, thus

$$e_{step}(\infty) = \frac{1}{1 + K_p}$$

2. Velocity Constant: $K_v = \lim_{s \rightarrow 0} sG(s)$, thus

$$e_{ramp}(\infty) = \frac{1}{K_v}$$

3. Acceleration Constant: $K_a = \lim_{s \rightarrow 0} s^2 G(s)$, thus

$$e_{parabola}(\infty) = \frac{1}{K_a}$$

System Type

- ▶ The static error constants are determined by the structure of $G(s)$.
- ▶ They are mostly determined by the number of integrators in $G(s)$.
- ▶ The **system type** is the number of integrators in the forward path, thus the value of n in figure below.

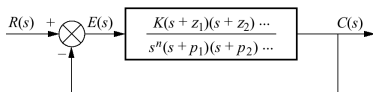


Figure 7.8.

Steady-State Error Summary

- Table shows relationship between input type, system type, static error constants, and steady-state errors.

Input	Steady-state error formula	Type 0		Type 1		Type 2	
		Static error constant	Error	Static error constant	Error	Static error constant	Error
Step, $u(t)$	$\frac{1}{1 + K_p}$	$K_p = \text{Constant}$	$\frac{1}{1 + K_p}$	$K_p = \infty$	0	$K_p = \infty$	0
Ramp, $tu(t)$	$\frac{1}{K_v}$	$K_v = 0$	∞	$K_v = \text{Constant}$	$\frac{1}{K_v}$	$K_v = \infty$	0
Parabola, $\frac{1}{2}t^2u(t)$	$\frac{1}{K_a}$	$K_a = 0$	∞	$K_a = 0$	∞	$K_a = \text{Constant}$	$\frac{1}{K_a}$

Table

7.2.

Tight Steady-State Error Specifications

- ▶ Example of a system requiring tight steady-state error specifications to be useful.



Figure 7.9.

Steady-State Error Specifications eg.

- For system below, find value of K such there is 10% error in steady state.

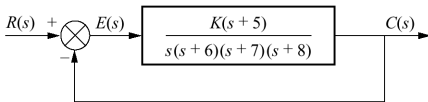


Figure 7.10.

Steady-State Error and Disturbances

- ▶ Can use feedback systems to handle unwanted disturbances to the systems.
- ▶ By using feedback, we can design systems that follow the input signal with small or zero error, *despite* these disturbances.
- ▶ Consider feedback system below with disturbance, $D(s)$, added between plant and controller.
- ▶ The system output is

$$C(s) = E(s)G_1(s)G_2(s) + D(s)G_2(s) \quad (26)$$

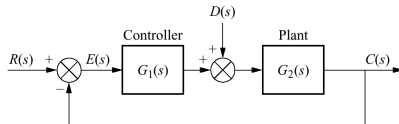


Figure 7.11.

Steady-State Error and Disturbances - II

- ▶ However

$$E(s) = R(s) - C(s) \Rightarrow C(s) = R(s) - E(s) \quad (27)$$

- ▶ Using Equations 27 and 26 and solving for $E(s)$ gives

$$E(s) = \frac{R(s)}{1 + G_1(s)G_2(s)} - \frac{D(s)G_2(s)}{1 + G_1(s)G_2(s)} \quad (28)$$

- ▶ Using final-value theorem, the steady-state error is

$$e_{ss} = \lim_{s \rightarrow 0} sE(s) \quad (29)$$

$$= \lim_{s \rightarrow 0} \frac{sR(s)}{1 + G_1(s)G_2(s)} - \lim_{s \rightarrow 0} \frac{sD(s)G_2(s)}{1 + G_1(s)G_2(s)} \quad (30)$$

$$= e_R(\infty) + e_D(\infty) \quad (31)$$

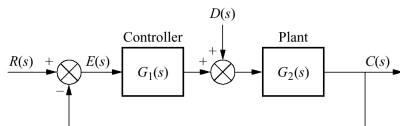


Figure 7.11.

Steady-State Error and Disturbances - III

- ▶ The $e_R(\infty)$ term is the steady-state error due to input $R(s)$ that we have already seen.
- ▶ The $e_D(\infty)$ term is the steady-state error due to $D(s)$.
- ▶ If $D(s) = 1/s$ (step input), we have

$$e_D(\infty) = -\frac{1}{\lim_{s \rightarrow 0} \frac{1}{G_2(s)} + \lim_{s \rightarrow 0} G_1(s)} \quad (32)$$

- ▶ If we set $R(s) = 0$, we get from Eqn28 the transfer function:

$$\frac{E(s)}{D(s)} = -\frac{G_2(s)}{1 + G_1(s)G_2(s)} \quad (33)$$

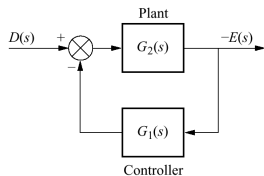


Figure 7.12.

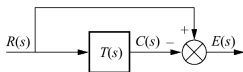
Steady-State Error and State Space

- ▶ We now consider how to evaluate steady-state error for a system represented in state-space.
- ▶ As we saw in Section 3.6 of the text, we can convert a single-input single-output state-space representation to an equivalent closed-loop transfer function using

$$T(s) = \frac{Y(s)}{U(s)} = \underline{C}(s\underline{I} - \underline{A})^{-1}\underline{B} \quad (34)$$

- ▶ In Diagram below, we have $E(s) = R(s) - C(s)$.
- ▶ We also have:

$$C(s) = R(s)T(s) \quad (35)$$



(a)

Figure 7.3.

Steady-State Error and State Space - II

- ▶ Combining the two we get

$$E(s) = R(s) - R(s)T(s) = R(s)[1 - T(s)] \quad (36)$$

- ▶ We thus have

$$\begin{aligned} e_{ss} &= \lim_{s \rightarrow 0} s E(s) \\ &= \lim_{s \rightarrow 0} s R(s)[1 - T(s)] \end{aligned} \quad (37)$$

- ▶ Substituting in for $T(s)$ gives

$$e_{ss} = \lim_{s \rightarrow 0} s R(s)[1 - \underline{C}(s\underline{I} - \underline{A})^{-1}\underline{B}] \quad (38)$$