



a) Turbine

state 1 state 2

$$T_1 = 400^\circ\text{C} = 673\text{K} \quad T_2 = 100^\circ\text{C} = 373\text{K}$$

$$P_1 = ? \quad P_2 = 100\text{ kPa}$$

$$V_1 = 20\text{ m/s} \quad V_2 = 5\text{ m/s}$$

$$A_1 = 0.25\text{ m}^2$$

$$\begin{aligned} E_{out} &= E_{in} \\ (E_{out} + KE_{out} + H_{out}) &= KE_{in} + H_{in} \\ -W_{out} &= \Delta H + \Delta KE \end{aligned}$$

$$-W_{out} = \dot{m}(\Delta h + \Delta ke)$$

$$\therefore 300 \leq \Delta T \leq 500 \rightarrow \Delta h = C_p(T_2 - T_1) \quad (\text{Table A2b})$$

$$\frac{C_p - 1.029}{523 - 500} = \frac{1.040 - 1.029}{550 - 500}$$

$$C_p = 1.034 \text{ kJ/kg}\cdot\text{K}$$

$$P_{1A} V_{1A} = R_{air} T_{1A}$$

$$\therefore = P_{...} T / D$$

$$P_{IA} V_{IA} = K_{air} T_{IA}$$

$$V_{IA} = R_{air} T_{IA} / P_{IA}$$

$$\dot{m}_A = \frac{A_{IA} V_{IA}^0}{V_{IA}} = \frac{P_{IA} A_{IA} V_{IA}^0}{R_{air} T_{IA}}$$

$$(\dot{\omega}_{out})_A = - \frac{P_{IA} A_{IA} V_{IA}^0}{R_{air} T_{IA}} \left(C_p (T_{2A} - T_{IA}) + \frac{1}{2} \frac{(V_{2A}^0 - V_{IA}^0)^2}{1000} \right) \quad (1)$$

Compressor

<u>State 1</u>	<u>State 2</u>	@ 100 kPa, $T_1 > T_{sat}$ \Rightarrow superheated
$T_{1B} = 150^\circ\text{C}$	$T_{2B} = 500^\circ\text{C}$	$h_{1B} = 2776.6 \text{ kJ/kg}$ from Table A6
$P_{1B} = 100 \text{ kPa}$	$P_{2B} = 2 \text{ MPa}$	
$V_{1B}^0 = 0 \text{ m/s}$	$V_{2B}^0 = 100 \text{ m/s}$	@ 2 MPa, $T_{2B} > T_{sat}$
$\dot{m}_B = 10 \text{ kg/s}$		$h_{2B} = 3468.3 \text{ kJ/kg}$ from Table A6

$$\bar{E}_{in} = E_{out}$$

$$H_{in} + KE_{in} + \dot{W}_{in} = H_{out} + KE_{out}$$

$$(\dot{\omega}_{in})_B = \dot{m}_B [(h_{2B} - h_{1B}) + \frac{1}{2} (V_{2B}^0 - V_{1B}^0)^2]$$

$$(\dot{\omega}_{in})_B = \dot{m}_B \left((h_{2B} - h_{1B}) + \frac{1}{2} \frac{(V_{2B}^0 - V_{1B}^0)^2}{1000} \right) \quad (2)$$

$$\cancel{(\dot{\omega}_{in})_B = (\dot{\omega}_{out})_A = \dot{\omega}}$$

$$\dot{\omega} = \dot{m}_B \left((h_{2B} - h_{1B}) + \frac{1}{2} \frac{(V_{2B}^0 - V_{1B}^0)^2}{1000} \right) = - \frac{P_{IA} A_{IA} V_{IA}^0}{R_{air} T_{IA}} \left(C_p (T_{2A} - T_{IA}) + \frac{1}{2} \frac{(V_{2A}^0 - V_{IA}^0)^2}{1000} \right)$$

$$\dot{\omega} = (10) \left((3468.3 - 2776.6) + \frac{1}{2} \frac{(100^2 - 0^2)}{1000} \right)$$

$$\boxed{\dot{\omega} = 6967 \text{ kW}}$$

$$\dot{h}_{in} = - P_{IA} A_{IA} V_{IA}^0 / (r_{air} T_{IA} + \frac{1}{2} (V_{IA}^0)^2 / (1000))$$

$$b) \ddot{\omega} = -\frac{P_{1A} A_{1A} V_{1A}^0}{R_{air} T_{1A}} \left(C_p (T_{2A} - T_{1A}) + \frac{1}{2} \frac{(V_{2A}^0 - V_{1A}^0)^2}{1000} \right)$$

$$6967 = -\frac{P_{1A}(0.25)(20)}{(0.287)(400+273)} \left[1.034(100-400) + \frac{1}{2000} (5^2 - 20^2) \right]$$

$$6967 = P_{1A}(8.0348)$$

$$\boxed{P_{1A} = 867.099 \text{ kPa}}$$

$$c) \dot{m}_B = \frac{A_{2B} V_{2B}^0}{V_{2B}}$$

$$A_{2B} = \frac{\dot{m}_B V_{2B}}{V_{2B}^0} = \frac{(10) V_{2B}}{100} = \frac{V_{2B}}{10}$$

@ $P_{2B} = 2 \text{ MPa}, 500^\circ \text{C} > T_{\text{sat}} \therefore \text{superheated}$

$$V_{2B} = 0.17568 \text{ m}^3/\text{kg}$$

$$A_{2B} = 0.17568 / 10 = \boxed{0.017568 \text{ m}^2}$$

$$d) V_{1A} = RT_{1A}/P_{1A} = (0.287)(400+273)/867.099$$

$$V_{1A} = 0.2228 \text{ m}^3/\text{kg}$$

$$V_{2A} = RT_{2A}/P_{2A} = (0.287)(100+273)/100$$

$$V_{2A} = 1.0705 \text{ m}^3/\text{kg}$$

Turbine

Compressor

