10 **1.a)** A load is to be moved at 2 m/s using a rack and pinion. If the motor speed is 1200 rpm, determine the pinion's required pitch radius.

Answer to a):

$$l = (2\pi/rev)^{r_p}$$

$$V = lw$$

$$V = (2\pi/rev)^{r_pw}$$

$$V = (2\pi/rev)^{r_pw}$$

$$V = (2\pi/rev)^{r_pw} = \frac{2 m/s}{(2\pi/rev)(1200 \frac{vev}{min})(\frac{1 min}{60s})} = 0.0159 m$$

**b)** For the same rack and pinion, if a force of 400 N is required to move the load, and the rack and pinion's efficiency is 0.8, determine the required motor torque.

Answer to b):

Fout = 
$$\frac{T_{in}}{r_{p}} h_{r}$$
  
 $\therefore T_{in} = \frac{F_{out} r_{p}}{n_{rp}} = \frac{(400 \text{ N})(0.0159 \text{ m})}{0.8} = 7.95 \text{ Nm}$ 

c) Now, the load is to be moved at 0.5 m/s using a lead screw. If the screw has a lead of 0.005 m/rev, determine the required motor speed in rpm.

Answer to c):

$$v = lw$$
 $v = lw = \frac{(0.5 \text{ m/s})(\frac{60 \text{ s}}{1 \text{ min}})}{0.005 \text{ m/rev}} = 6000 \text{ rpm}$ 

d) For the same lead screw, if a force of 400 N is required to move the load and the screw's efficiency is 0.6, determine the required motor torque.

Answer to d):

$$T = \frac{Fl}{(2\pi/rev)\eta_s} = \frac{(400 \text{ N})(0.005 \text{ m/rev})}{(2\pi/rev)0.6} = 0.53 \text{ Nm}$$

- 2. A single rod pneumatic cylinder will be used to move a mass horizontally in both directions at a speed of 0.2 m/s. The cylinder must overcome a friction force of 2000 N during its motion. The inertia force is relatively small and can be neglected. The rod's cross-sectional area is  $9 \times 10^{-4}$  m<sup>2</sup>, the supply pressure is  $6 \times 10^5$  Pa gauge and the air temperature is 25 °C. If a pressure drop of  $4 \times 10^4$  Pa across the valve is desired, determine:
  - a) The minimum bore cross-sectional area required.
  - b) The minimum valve flow coefficient required.

Assume that the pressure drop across the valve is the same for the return flow as for

$$\frac{P_{supply} - DIJArod}{P_{supply} - 2DP}$$
=  $\frac{2000 \text{ N} + (6 \times 10^5 \text{ Pa} - 4 \times 10^4 \text{ Pa})(9 \times 10^{-4} \text{ m}^2)}{6 \times 10^5 \text{ Pa} - 2(4 \times 10^4 \text{ Pa})}$ 

b) Use extend dir'n to get min (v with single rod cylinder)
$$P = \frac{P_1 - DP}{R_S T} = \frac{(6 \times 10^5 \, Pa + 1.01 \times 10^5 \, Pa) - 4 \times 10^4 \, Pa}{(287 \, \text{J/kgK}) (25 + 273) \, \text{K}}$$

$$= 7.73 \, \text{kg/m}^3$$

 $Q_{\text{max}} = V_{\text{max}} A_{\text{bore}} = (0.2 \text{ m/s})(0.0048 \text{ m}^2) = 9.63 \times 10^{-4} \text{ m}^3/\text{s}$ 

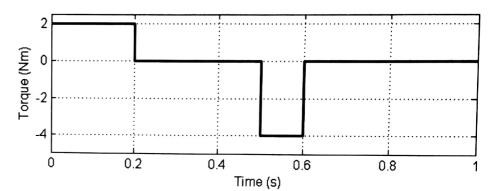
$$C_{V} = (4.22 \times 10^{4} \,\mathrm{m}^{-2}) \, \Omega_{max} \int \frac{e}{DP}$$

$$= (4.22 \times 10^{4} \,\mathrm{m}^{2}) \, (9.63 \times 10^{-4} \,\mathrm{m}^{3}/\mathrm{s}) \int \frac{7.73 \,\mathrm{kg/m}^{3}}{4 \times 10^{4} \,\mathrm{Pa}}$$

$$= 0.56$$

9 3. A motion cycle requires a motor to produce the torque profile shown below. The ambient temperature is 30 °C and the motor's parameters are given in the table. Determine the temperature the motor windings will reach if the motion cycle is repeated continuously.

Torque constant	0.4
(Nm/A)	
Resistance at max.	1.2
temp. (ohm)	
Total thermal	2
resistance (°C/W)	



$$T_{RMS} = \sqrt{\sum_{i=1}^{n} \tau^2 t_i / \sum_{i=1}^{n} t_i}$$

$$= \int \frac{(2 \text{ Nm})^2 (0.2s) + (-4 \text{ Nm})^2 (0.1s)}{0.2s + 0.3s + 0.1s + 0.4s} = 1.55 \text{ Nm}$$

$$I_{RMS} = \frac{T_{RMS}}{K_{t}} = \frac{1.55 \text{ Nm}}{0.4 \text{ Nm/A}} = 3.88 \text{ A}$$

$$P_{5} = I_{RMS}^{2} R_{Hot} = (3.88 A)^{2} (1.2 ohm) = 18 W$$

$$T_{W} = T_{a} + P_{r}R_{+h}$$
  
= 30°C + (18 W)(2°C/W)  
= 66°C

4. A linear actuator consisting of a motor, gearbox and ball screw moves a load vertically. A positive motor velocity moves the load upwards. The screw's moment of inertia is 1.5 × 10<sup>-4</sup> kgm<sup>2</sup>, the motor's moment of inertia is 3 × 10<sup>-5</sup> kgm<sup>2</sup> and the gear ratio is 2.5. The screw's lead is 0.002 m/rev. If the load has a mass of 500 kg and is subject to a 800 N friction force, determine the motor torque required for the load to accelerate upwards at 1.2 m/s<sup>2</sup>. The friction of the gears and ball screw can be neglected.

Fext = mg + F<sub>4</sub>

$$= (500 \text{ kg})(9.81 \text{ N/kg}) + 800 \text{ N}$$

$$= 5705 \text{ N}$$

$$\text{Text} = \frac{\text{Fextl}}{2\pi/\text{rev}} = \frac{(5705 \text{ N})(0.002 \text{ m/rev})}{2\pi/\text{rev}} = 1.82 \text{ Nm}$$

$$J_{load} = m \left(\frac{l}{2\pi}\right)^{2} + J_{screw}$$

$$= (500 \text{ kg}) \left(\frac{0.002 \text{ m/rev}}{2\pi/\text{rev}}\right)^{2} + 1.5 \times 10^{-4} \text{ kgm}^{2} = 2.01 \times 10^{-4} \text{ kg m}^{2}$$

$$\dot{w}_{load} = \frac{\alpha}{l} = \frac{1.2 \text{ m/s}^2}{0.002 \text{ m/rev}} = (600 \text{ rev/s}^2) \left(\frac{2\pi}{\text{rev}}\right) = 3.77 \times 10^3 \frac{\text{rad}}{\text{s}^2}$$

$$\dot{w}_{\text{motor}} = N_r \dot{w}_{\text{load}} = (2.5)(3.77 \times 10^3 \, \text{rad/s}^2) = 9.42 \times 10^3 \, \text{rad/s}^2$$

$$T_{\text{motor}} = \left(J_{\text{motor}} + \frac{1}{N_r^2} J_{\text{load}}\right) \dot{w}_{\text{motor}} + \frac{1}{N_r} T_{\text{ex}} + \frac{1}{N_r} T_{\text{ex}}$$

$$= \left(3 \times 10^{-5} \, kg m^2 + \left(\frac{1}{2.5^2}\right) \left(2.01 \times 10^{-4} \, kg m^2\right)\right) 9.42 \times 10^3 \, rad/s^2 + \left(\frac{1}{2.5}\right) \left(1.82 \, Nm\right)$$