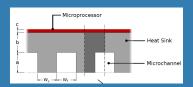


# Unstructured FEM: 2D Unsteady Heat Diffusion







Abhishek Y. Deshmukh, Raghavan Lakshmanan, Mohsin Ali Chaudry, A. Emre Ongut January 31, 2014





Verification

Objective

Optimization

Finite Element Method

**Parallel Computing** 

**FEM Formulation** 

Application

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Summary







# **Objective**

- To develop a parallel finite element solver that uses unstructured grid to solve a 2D transient heat diffusion problem.
- Application to a realistic engineering heat transfer problem. Heat diffusion in Microchannels in the heat sink of microprocessors.



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## **General Steps**

#### Preprocessing

- Define a domain of interest
- Discretization of domain into finite elements

#### Solution

- Define shape functions to represent the behaviour of elements
- Construct local stiffness matrix and then ultimately global stiffness matrix
- Apply boundary condition, initial conditions and forcing function
- Solving linear system of algebraic equations

#### Postprocessing

Analyse the solution







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## **2D Unsteady Heat Diffusion Equation**

$$\frac{\partial T}{\partial t} - \alpha \nabla^2 T = f, \text{ in } \Omega \in \Re^2$$

where  $\alpha$  is diffusion coefficient.

$$\alpha = \frac{k}{\rho c_p}$$

k is heat conductivity,  $\rho$  is the density and  $c_p$  is the specific heat capacity of the material.

$$f = \frac{\dot{q}}{\rho c_p}$$



# **Initial and Boundary Conditions**

- Intial condition:  $T(x, y, 0) = T_0(x, y)$
- Boundary conditions
  - · Dirichlet:

$$T(x,0,t) = T_1, T(x,L,t) = T_2, T(0,y,t) = T_3, T(L,y,t) = T_4$$

Neumann:

$$k\left(n_{x}\frac{\partial T}{\partial x}+n_{y}\frac{\partial T}{\partial y}\right)=q_{0}$$

· Robin (Mixed):

$$h(T - T_{inf}) = -k \left( n_x \frac{\partial T}{\partial x} + n_y \frac{\partial T}{\partial y} \right)$$







## Weighted Residual Method

$$\int_{\Omega} (w \frac{\partial T}{\partial t} + \alpha w \nabla^2 T) d\Omega = \int_{\Omega} w f d\Omega$$

where w is a weighting function or test function. Integrating by parts,

$$\int_{\Omega} (w \frac{\partial T}{\partial t} + \alpha \nabla w \cdot \nabla T) d\Omega = \int_{\Omega} w f d\Omega + \frac{1}{\rho c_{\rho}} \int_{\Gamma} w \underbrace{(k \hat{n} \cdot \nabla T)}_{k(n_{x} \frac{\partial T}{\partial x} + n_{y} \frac{\partial T}{\partial y})} d\Gamma$$



## Approximation

- Approximate the temperature as a piecewise linear combination of shape functions  $S_i(x, y)$ .
- These shape functions remain constant over time. The coefficients  $T_i(t)$  are time dependent.

$$T(x,y,t) = \sum_{j=1}^{nn} T_j(t)S_j(x,y)$$

Galerkin Finite Element Method:

$$w = S_i(x)$$



## Approximation [contd.]

$$\underbrace{\sum_{j=1}^{nn} \int_{\Omega} (S_{i}S_{j}) d\Omega \frac{\partial T_{j}}{\partial t}}_{[M][\bar{T}]} - \underbrace{\sum_{j=1}^{nn} \left[ \int_{\Omega} \alpha \left( \frac{\partial S_{i}}{\partial x} \frac{\partial S_{j}}{\partial x} + \frac{\partial S_{i}}{\partial y} \frac{\partial S_{j}}{\partial y} \right) d\Omega \right] T_{j}}_{[K][T]}$$

$$= \underbrace{\int_{\Omega} S_{i} f d\Omega}_{[F]} + \underbrace{\frac{1}{\rho c_{p}} \int_{\Gamma} S_{i}(k\hat{n}.\nabla T) d\Gamma}_{[F]}$$

$$i, j = 1, 2, ..., nn$$

$$[K]_{ij} = \int_{\Omega} \alpha \left( \frac{\partial S_i}{\partial x} \frac{\partial S_j}{\partial x} + \frac{\partial S_i}{\partial y} \frac{\partial S_j}{\partial y} \right) d\Omega$$























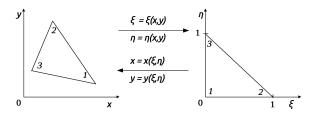








#### **Triangular Elements**



Co-ordinate transformation

#### Shape functions:

$$S_1(\xi, \eta) = 1 - \xi - \eta$$
,  $S_2(\xi, \eta) = \xi$ ,  $S_3(\xi, \eta) = \eta$ 







$$\begin{split} \frac{\partial S_{1}}{\partial \xi} &= -1, \frac{\partial S_{2}}{\partial \xi} = 1, \frac{\partial S_{3}}{\partial \xi} = 0, \frac{\partial S_{1}}{\partial \eta} = -1, \frac{\partial S_{2}}{\partial \eta} = 0, \frac{\partial S_{3}}{\partial \eta} = 1\\ \frac{\partial S_{i}}{\partial \xi} &= \frac{\partial S_{i}}{\partial x} \frac{\partial x}{\partial \xi} + \frac{\partial S_{i}}{\partial y} \frac{\partial y}{\partial \xi}, \frac{\partial S_{i}}{\partial \eta} = \frac{\partial S_{i}}{\partial x} \frac{\partial x}{\partial \eta} + \frac{\partial S_{i}}{\partial y} \frac{\partial y}{\partial \eta}\\ & \left[ \frac{\partial S_{i}}{\partial \xi} \right] = \underbrace{ \begin{bmatrix} \frac{\partial x}{\partial \xi} & \frac{\partial y}{\partial \xi} \\ \frac{\partial x}{\partial \eta} & \frac{\partial y}{\partial \eta} \end{bmatrix} }_{Jacobian} \begin{bmatrix} \frac{\partial S_{i}}{\partial x} \\ \frac{\partial S_{i}}{\partial y} \end{bmatrix} \end{split}$$



#### Isoparametric formulation:

$$x(\xi,\eta) = \sum_{j=1}^{nen} x_j^e S_j(\xi,\eta)$$
,  $y(\xi,\eta) = \sum_{j=1}^{nen} y_j^e S_j(\xi,\eta)$  Jacobian:

$$\begin{bmatrix} J \end{bmatrix} = \begin{bmatrix} \frac{\partial S_1}{\partial \xi} & \frac{\partial S_2}{\partial \xi} & \frac{\partial S_3}{\partial \xi} \\ & & & \\ \frac{\partial S_1}{\partial \eta} & \frac{\partial S_2}{\partial \eta} & \frac{\partial S_3}{\partial \eta} \end{bmatrix} \begin{bmatrix} x_1^e & y_1^e \\ x_2^e & y_2^e \\ x_3^e & y_3^e \end{bmatrix} = \begin{bmatrix} \frac{\partial x}{\partial \xi} & \frac{\partial y}{\partial \xi} \\ \frac{\partial x}{\partial \eta} & \frac{\partial y}{\partial \eta} \end{bmatrix}$$

But we need  $[J]^{-1}$  which is the inverse mapping.

$$\begin{bmatrix} \frac{\partial S_i}{\partial x} \\ \frac{\partial S_i}{\partial y} \end{bmatrix} = \underbrace{\begin{bmatrix} \frac{\partial \xi}{\partial x} & \frac{\partial \eta}{\partial x} \\ \frac{\partial \xi}{\partial y} & \frac{\partial \eta}{\partial y} \end{bmatrix}}_{\text{I.II-1}} \underbrace{\begin{bmatrix} \frac{\partial S_i}{\partial \xi} \\ \frac{\partial S_i}{\partial \eta} \end{bmatrix}}_{\text{known}}$$





Calculation of element level stiffness matix:

$$K_{ij}^e = \int_{\Omega_e} \alpha (\frac{\partial S_i}{\partial x} \frac{\partial S_j}{\partial x} + \frac{\partial S_i}{\partial y} \frac{\partial S_j}{\partial y}) dx dy \text{ i,j = 1,2,...,nen}$$

Integral transformation to master element:

$$K_{ij}^{e} = \int_{0}^{1} \int_{0}^{1-\xi} \alpha [f_{k}(\xi, \eta)] |J| d\xi d\eta$$

$$f_{k}(\xi, \eta) = \left( \frac{\partial S_{i}}{\partial \xi} \frac{\partial \xi}{\partial x} + \frac{\partial S_{i}}{\partial \eta} \frac{\partial \eta}{\partial x} \right) \left( \frac{\partial S_{j}}{\partial \xi} \frac{\partial \xi}{\partial x} + \frac{\partial S_{j}}{\partial \eta} \frac{\partial \eta}{\partial x} \right)$$

$$+ \left( \frac{\partial S_{i}}{\partial \xi} \frac{\partial \xi}{\partial y} + \frac{\partial S_{i}}{\partial \eta} \frac{\partial \eta}{\partial y} \right) \left( \frac{\partial S_{j}}{\partial \xi} \frac{\partial \xi}{\partial y} + \frac{\partial S_{j}}{\partial \eta} \frac{\partial \eta}{\partial y} \right)$$

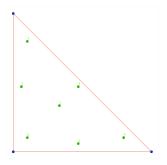
i, j = 1, 2, ..., nen

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#### Numerical Integration using 7 point Gauss Quadrature rule:



$$K_{ij}^e \simeq \sum_{l=1}^{nGQP} f_k(\xi_l, \eta_l) w_l$$





#### Solution

In matrix form :  $[M][\dot{T}] + [K][T] = [F] + [B]$ Forward Euler explicit differencing in time:

$$\frac{dT}{dt}|_{s} = \frac{T^{s+1} - T^{s}}{\Delta t} + O(\Delta t)$$

$$[M] \frac{[T]^{s+1} - [T]^{s}}{\Delta t} + [K][T]^{s} = [F] + [B]$$

$$[M][T]^{s+1} = [M][T]^{s} + \Delta t([F] + [B] - [K][T]^{s})$$

$$[M][T]^{s+1} = [RHS]^{s}$$





## **Lumping of Mass Matrix**

The inversion of Mass matrix can be avoided if we do a physical approximation called lumping of mass matrix.

Consistent element mass matrix:  $[M_{cii}]$ 

Lumped element mass matrix:  $[M_{Lii}]$ 

$$M_{Lii} = M_{cii} \frac{\sum_{i=1}^{n} \sum_{j=1}^{n} M_{cij}}{\sum_{j=1}^{n} M_{cjj}}$$

This is correct only for linear elements.  $[M_{Lii}]$  is a diagonal matrix.



## **Boundary Integral**

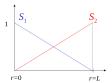
Dirichlet:  $[B]_i = T_w$ ,  $[K]_{ii} = 1$ ,  $[K]_{ii} = 0 \ \forall i \neq j$ For example, if for some element, node temperatures T1 and T3 are known then [B] and [K] for that element can be written as:

$$[B] = \begin{bmatrix} T_w \\ B_2 \\ T_w \end{bmatrix} \text{ and } [K] = \begin{bmatrix} 1 & 0 & 0 \\ * & * & * \\ 0 & 0 & 1 \end{bmatrix}$$

\* indicate non-zero values.



Neumann (Flux Variable(FV) =  $q_{in}$ ):



$$B_{1}^{e} = \frac{1}{\rho c_{p}} \int_{\Gamma} S_{1}(FV) d\Gamma = \frac{1}{\rho c_{p}} \int_{0}^{L} (1 - \frac{r}{L}) q_{in} dr = \frac{q_{in} L}{2\rho c_{p}}$$

$$B_{2}^{e} = \frac{1}{\rho c_{p}} \int_{\Gamma} S_{2}(FV) d\Gamma = \frac{1}{\rho c_{p}} \int_{0}^{L} \frac{r}{L} q_{in} dr = \frac{q_{in} L}{2\rho c_{p}}$$

$$[B] = \begin{bmatrix} \frac{q_{in} L}{2\rho c_{p}} \\ \frac{q_{in} L}{2\rho c_{p}} \\ 0 \end{bmatrix}$$



Robin(Mixed type): Flux variable (FV) can be expressed as

$$-k\hat{n}.\nabla T=h(T-T_{\infty})$$

FV = aPV + b where a = -h and b =  $hT_{\infty}$ 

$$B_1^e = \frac{1}{\rho c_p} \int_{\Gamma} S_1(FV) d\Gamma = \frac{1}{\rho c_p} \int_{0}^{L} (1 - \frac{r}{L}) (aPV + b) dr$$

$$B_2^e = \frac{1}{\rho c_p} \int_{\Gamma} S_2(FV) d\Gamma = \frac{1}{\rho c_p} \int_0^L \frac{r}{L} (aPV + b) dr$$

Primary variable (PV) temperature:





$$B_{1}^{e} = \frac{1}{\rho c_{p}} \int_{0}^{L} (1 - \frac{r}{L}) [a((1 - \frac{r}{L})T_{1} + \frac{r}{L}T_{2}) + b] dr$$

$$= \frac{L}{6\rho c_{p}} (3b + 2aT_{1} + aT_{2})$$

$$B_{2}^{e} = \frac{1}{\rho c_{p}} \int_{0}^{L} \frac{r}{L} [a((1 - \frac{r}{L})T_{1} + \frac{r}{L}T_{2}) + b] dr$$

$$= \frac{L}{6\rho c_{p}} (3b + aT_{1} + 2aT_{2})$$





Element boundary vector can be written as:

$$[B] = egin{bmatrix} rac{L}{6
ho c_{
ho}}(3b + 2aT_{1} + aT_{2}) \ rac{L}{6
ho c_{
ho}}(3b + aT_{1} + 2aT_{2}) \ 0 \end{bmatrix}$$

Coefficients of  $T_1$  and  $T_2$  transferrd to K. Modified [B] and corresponding [K] can be written as:

$$[B] = \begin{bmatrix} \frac{Lb}{2\rho c_\rho} \\ \frac{Lb}{2\rho c_\rho} \\ 0 \end{bmatrix} \text{ and } [K] = \begin{bmatrix} (* - \frac{aL}{3\rho c_\rho}) & (* - \frac{aL}{6\rho c_\rho}) & * \\ (* - \frac{aL}{6\rho c_\rho}) & (* - \frac{aL}{3\rho c_\rho}) & * \\ * & * & * \end{bmatrix}$$



Implementation





## **Program Flow Chart**

- Preprocessing Stage:
  - Reading settings file
  - Reading mesh data
- 2 Solution Stage:
  - Calculating Jacobian matrix for all elements
  - Calculating element stiffness matrix for all elements
  - Applying boundary conditions
  - **Explicit Solver**

Input: Mesh files, Initial and Boundary conditions, Diffusion Coefficient, Source term. No. of iterations. Time step, Data Writing Frequency Define Mesh Structures Calculate Metrics: Jacobian Element level Matrices Apply Boundary Conditions Solver: Solve 2D Heat diffusion equation Write time+=dtVTK Data iter<NIter no Stop

Start











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#### Case 1: 1D

A simple 1D case is taken as an example. Domain:  $x \in [0,L]$ L=2.

Initial condition:  $T_i = T(x,0) = 1000K$ 

Boundary Conditions:  $T_L = T(0,t) = 0$ ,  $T_R = T(L,t) = 0$ ,

 $T_{top}$  and  $T_{bottom}$  are insulated

Analytical Solution:

$$T(x,t) = \sum_{n=1}^{\infty} B_n sin(\frac{n\pi x}{L}) e^{-\frac{n^2 \pi^2 \alpha t}{L^2}}$$

where  $\alpha$  is diffusion coefficient and

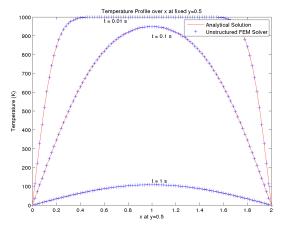
$$B_n = -T_i \frac{2(-1+(-1)^n)}{n\pi}$$







#### **Comparison with Analytical solution**







## Case 2: 2D Quenching of a Billet[1]

Domain: A long billet of rectangular cross section  $2a \times 2b$ (a=2, b=1). Because of symmetry, analyze one quarter of the cross section

Shifted temperature:  $\theta(x, y, t) = T(x, y, t) - T_{\infty}$ Material properties:  $\alpha = 1$ ,  $\rho = 1$ ,  $c_p = 1$ , h = 1

Initial condition:  $\theta(x, y, 0) = T_i - T_{\infty} = \theta_i$ ,  $T_i = 1000$ ,

 $T_{\infty} = 300$ 

**Boundary Conditions:** 

$$x = 0$$
:  $\frac{\partial \theta}{\partial x} = 0$   $y = 0$ :  $\frac{\partial \theta}{\partial y} = 0$   $x = a$ :  $\frac{\partial \theta}{\partial x} = -\frac{h}{k}\theta(a, y, t)$   $y = b$ :  $\frac{\partial \theta}{\partial y} = -\frac{h}{k}\theta(x, b, t)$ 





#### Analytical solution:

$$\frac{\theta}{\theta_i} = 4 \sum_{n=1}^{\infty} \sum_{m=1}^{\infty} e^{(-\alpha(\lambda_n^2 + \beta_m^2)t)} \times$$

$$\frac{sin(\lambda_n a)cos(\lambda_n x)sin(\beta_m b)cos(\beta_m y)}{[\lambda_n a + sin(\lambda_n a)cos(\lambda_n a)][\beta_m b + sin(\beta_m b)cos(\beta_m b)]}$$

where  $\alpha$  is diffusion coefficient and the eigenvalues  $\lambda_n$  and  $\beta_m$  are the roots of trancendental equations.

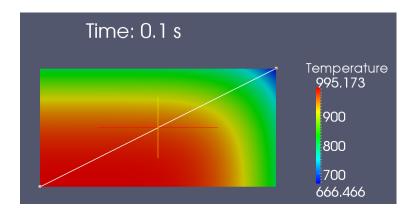
$$\lambda_n tan(\lambda_n a) = \frac{h}{k}$$
  $\beta_m tan(\beta_m b) = \frac{h}{k}$ 

where k is conductivity of the billet matrial.





#### Results

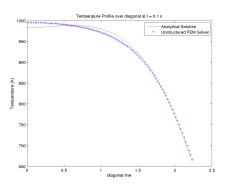


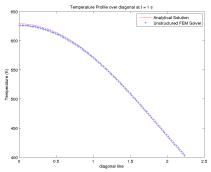






# Results [contd.]











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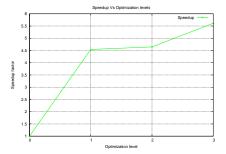






## **Optimization**

- Manual Optimization
  - · Local arrays and variables
  - Loop unrolling
- **Automatic Optimization** 
  - · 01
  - · O2
  - · O3







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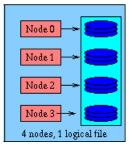
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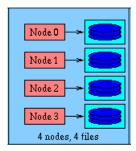




## **Parallelizing**



Parallel I/O 1 [3]



Parallel I/O 2 [3]





# **Structure of Mesh files (Illustration)**

	0	0
	0.5	0
	1	0
mxyz	2	0
	2	0.25
	2	0.5
	1.5	0.75
	1	0.75
	0.5	0.75
	0	0.5
	0	0.25
37 Jani	0.5	0.25

	8 9	11	10
mien	9	2 4	8
	1		7
	3	5	8
	3 5 4	11	10
		7	9
	6	8 3	3 9 2
	10	3	9
	7	1	
	5	4 9	8 7
	11	9	7

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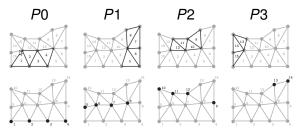






# First approach

All the node information not present on indvidual processors. Mesh data needs to be communicated before the actual FEM code starts. This adds to communication overhead.



Division of elements and nodes [4]

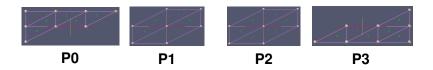






# Second approach

All mesh data present on the same processor.

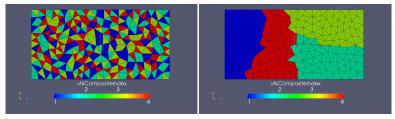


A preprocessor is needed to generate the separate domains for each processor. Once the mesh is divided it can be reused in further simulations.



## **Mesh Partitioning**

Mesh partitioning tool needs to be used.



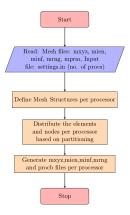
Before Mesh partitioning

After Mesh partitioning

Several public tools available, e.g. SCOTCH, CHACO, HARP, JOSTLE, METIS, etc. We used METIS.



## Parallelizing with second approach



#### Contents of a sample procb file

0	0	-1	-1	-1	-1
1	0	-1	-1	-1	-1
2	0	-1	-1	-1	-1
3	1	1	-1	-1	-1
13	1	1	-1	-1	-1
14	1	1	-1	-1	-1
15	2	1	2	-1	-1
16	2	1	2	-1	-1

#### Preprocessor Flow Chart

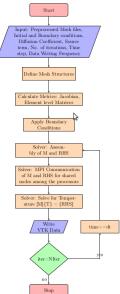






## **Program Flow Chart**

- 1 Preprocessing Stage:
  - Reading settings and mesh data
- 2 Solution Stage:
  - Calculating mesh metrics
  - Applying boundary conditions
  - Assembly of [M] and {RHS}
  - Communicate [M] and {RHS} for boundary nodes
  - Solve [M]{T} = {RHS}
- Post Processing Stage
  Jamery 31, 2014 Shifting Stage
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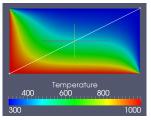






#### Verification of Parallel code

A rectangular domain is considered for heat diffusion problem. Left and bottom walls are kept at 1000 K while top and right walls are maintained at 300 K and the temperature profile is allowed to develop over time. The snapshots are taken at t=0.001s, t=0.01s and t=0.1s.



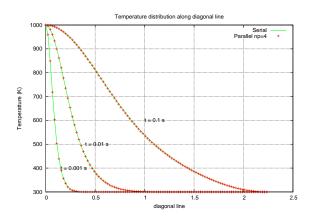
t = 0.1 s







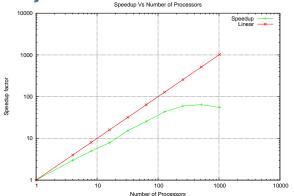
### **Verification of Parallel code**







# **Scalability**



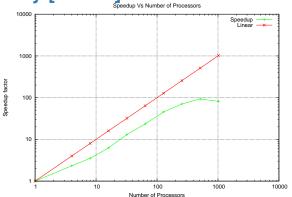
Finestmesh: ne = 516160, nn = 259082







Scalability [contd.]



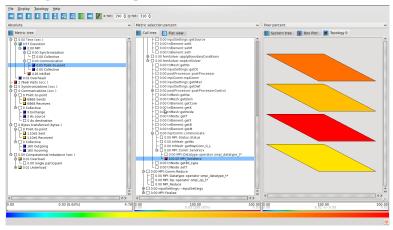
Superfine: ne = 945746, nn = 474074







## **Analysis using Scalasca**









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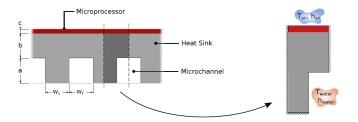
Summary







# **Application**



A heat sink for cooling computer chip, which is fabricated from copper with machined microchannels. Within these microchannels, water flows and carries away the heat dissipated by the chips.





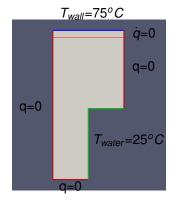


# Physical properties and dimensions:

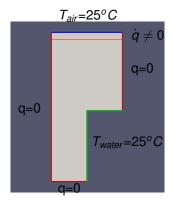
- Dimensions of the heat sink:  $a = b = w_s = w_f = 0.2$  mm, c = 0.02 mm
- Thermal diffusivity of copper:  $\alpha_{copper} = 1.11 \times 10^{-4} \ m^2/s$
- Density of copper:  $\rho = 8940 \ kg/m^3$
- Specific heat capacity:  $C_p = 385 \text{ J/kg/K}$
- Properties of cooling fluid:  $T_{water} = 25^{\circ}$  C,  $h_{water} = 30000$  $W/m^2/K$
- Properties of ambient air:  $T_{air} = 25^{\circ}$  C,  $h_{air} = 2 W/m^2/K$



Case 1: Steady state temperature distribution



Case 2: Find  $\dot{q}_{max}$  for  $T_{max} < 75^{\circ}C$ 

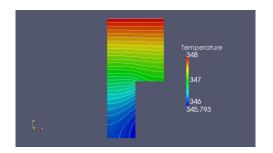






### Case 1

Steady state criteria: Maximum rate of change of temperature in the domain  $\left(\frac{dT}{dt}\right)_{max} \leq 0.001$ 



Time required to reach steady state = **8.4237117 ms** 

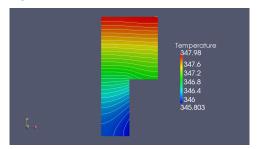






### Case 2

Use the result of case 1 as the initial condition. Maximum allowable temperature in the domain 75°C or 348 K



Maximum heat generation rate  $\approx$  **9.6**  $\times$  10<sup>10</sup>  $W/m^3$ . If we consider a chip of dimensions 10 mm $\times$ 10 mm  $\times$  0.02 mm, it is allowed to generate maximum 192 W.
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## **Summary**

- A two dimensional transient heat diffusion equation is solved numerically using finite element method on unstructured mesh.
- A serial object oriented C++ code is developed based on the FE formulation.
- The code is verified against analytical solutions found in [1].
- The code is parallelized to be able to solve larger problems faster.
- The parallel code shows sufficient amount of scalability.
- Finally, the capability of the software to be applied to a
   realistic problem has been demonstrated.
   January 31, 2014 Abhishek Y. Deshmukh, Radhavan Lakshmanan, Mohsin Ali Chaudry, A. Emre Ongut



### References

- Jordan Wall. Class materials for Conduction Heat Transfer. pages 5-8, 21.
- Class materials for Parallel Programming I.
- https://computing.llnl.gov/LCdocs/ioguide/.
- Class materials for Parallel Computing for Continuum Mechanics.



### Questions?



