A renowned laboratory reports quadruple-point coordinates of 10.2 Mbar and 24.1 deg C for the four-phase equilibrium of allotropic solid forms of the exotic chemical β -maiasmone. Examine the claim using the Gibbs phase rule and provide a plausible explanation for your results.

Solution

Something is wrong with the claim because it appears to violate the phase rule. The phase rule states that $F = 2 - \pi + N$. Since the number of phases is 4 and the number of chemically distinct species in 1, then F = 2 - 4 + 1 = -1. So either the number of chemically distinct species is greater than 1 or the number of phases is less than 4.

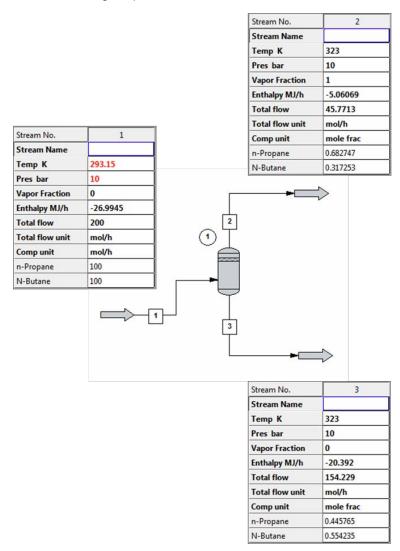
A system of propane and n-butane exists in two-phase vapor/liquid equilibrium at 10 bar and 323 K. The mole fraction of propane is about 0.67 in the vapor phase and about 0.40 in the liquid phase. Additional pure propane is added to the system, which is brought again to equilibrium at the same T and P, with both liquid and vapor phases still present. What is the effect of the addition of propane on the mole fractions of propane in the vapor and liquid phases?

Solution

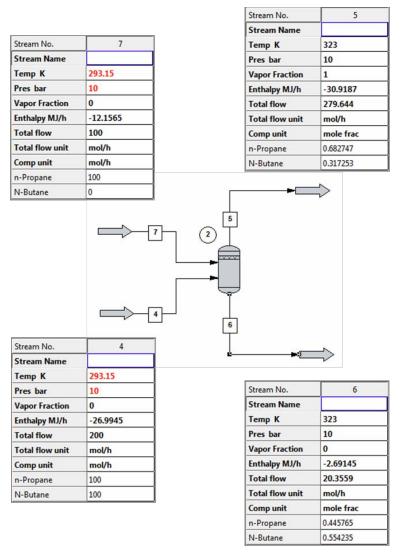
The phase rule states that $F = 2 - \pi + N = 2 - 2 + 2 = 2$, which means that the system has two degrees of freedom. Specifying T and P satisfies the degrees of freedom and fixes the *intensive* state of the system. As long as the system is returned to the same T and P, all intensive variables must also be the same. Since mole fractions are intensive properties, the phase rule predicts that the mole fractions must be unchanged. //ANS

This is illustrated in CHEMCAD below (not needed to answer the question).

Case 1: Flash at original specifications



Case 2: Adding more propane



Comparison of stream 2 with stream 5 shows that the mole fractions are the same. But something had to change. Notice that the total flows are different. Total flow is an extensive property.

The same observation results from comparing streams 3 and 6. The mole fractions are the same but the total flows are different.

(This will only be true as long as there are two phases!)

Express the volume expansivity (β) and isothermal compressibility (κ) as functions of density and its partial derivatives. For water at 50 degC and 1 bar, κ = 44.18 × 10⁻⁶ bar⁻¹. To what pressure must water be compressed at 50 degC to change its density by 1%? Assume κ is independent of P.

Solution

Part 1

Express β and κ as functions of density and its partial derivatives. The key to this part of the problem is to use equations 3.2-3.5 as a guide. In those equations, V=V(T,P). Just repeat the four equations using $\rho = \rho(T,P)$:

$$\begin{split} d\rho = & \left(\frac{\partial \rho}{\partial T}\right)_{P} dT + \left(\frac{\partial \rho}{\partial P}\right)_{T} dP \\ \beta \equiv & -\frac{1}{\rho} \left(\frac{\partial \rho}{\partial T}\right)_{P} \\ \kappa \equiv & +\frac{1}{\rho} \left(\frac{\partial \rho}{\partial P}\right)_{T} \\ \frac{d\rho}{\rho} = & -\beta dT + \kappa dP \end{split}$$

(*Cadets tend to put the wrong signs on β and $\kappa.*$)

We must use a negative sign in front of $\partial \rho/\partial T$ is negative, so the negative sign makes β positive. As temperature goes up, density goes down, so $\partial \rho/\partial T$ is negative. This property was originally derived from molar volume, where $\beta = (\partial V/\partial T)_P/V$. This derivative is always positive because molar volume increases as temperature increases.

When repeating the derivation with density, it is desirable to keep β the same since the property itself does not change, so we use the negative sign to compensate for the negative derivative.

Likewise, when derived from desnity, κ uses a positive sign in the deriviative because density increases as pressure increases, so $(\partial \rho/\partial P)_T$ is positive. In the original derivation from V, we used a negative sign because $(\partial V/\partial P)_T$ is negative. That is, the molar volume decreases as pressure increases.

Part 2

Calculate the pressure required to change the density by 1%

$$\kappa \equiv \frac{1}{\rho} \left(\frac{\partial \rho}{\partial P} \right)_T$$

$$\kappa \rho = \left(\frac{\partial \rho}{\partial P} \right)_T$$

$$\kappa \rho dP = d\rho \quad (constT)$$

$$\kappa dP = \frac{d\rho}{\rho} \quad (constT)$$

$$\int_{P1}^{P2} \kappa dP = \int_{\sigma1}^{\rho2} \frac{d\rho}{\rho} \quad (constT)$$

 $ln[*] = \rho 1 = 0.9881; (*0.9881 \frac{g}{cm^3})$ from engineering toolbox*)

$$In[*]:= \rho 2 = \rho 1 * 1.01;$$
 $\kappa = 44.18 * 10^{-6};$
 $P1 = 1;$
 $In[*]:= eq1 = \kappa * (P2 - P1) == Log \left[\frac{\rho 2}{\rho 1}\right];$
 $In[*]:= Solve[eq1, P2]$
 $Out[*]:= \left\{ \{P2 \rightarrow 226.222518179\} \right\}$
 $(* P2=226.223 bar //ANS*)$

For liquid water the isothermal compressibility is given by

$$K = \frac{c}{V(P+b)},$$

where c and b are functions of temperature only. If 1 kg of water is compressed isothermally and reversibly from 1 to 500 bar at 60 degC, how much work is required? At 60 degC, b=2,700 bar and c=0.125 $\rm cm^3$ g^{-1} .

Solution

The key to this problem is to start with the definition of κ given in equation 3.3, and use this equation to solve for the differential volume, and then substitute this into the definition of work in Equation 1.3:

$$\begin{split} \kappa &\equiv -\frac{1}{V} \bigg(\frac{\partial V}{\partial P} \bigg)_T \quad \text{and} \quad \kappa = \frac{c}{V \big(P + b \big)} \\ & \therefore \quad \frac{c}{V \big(P + b \big)} = -\frac{1}{V} \bigg(\frac{\partial V}{\partial P} \bigg)_T \\ & \qquad \frac{c}{P + b} = - \bigg(\frac{\partial V}{\partial P} \bigg)_T \\ & \qquad \text{d} V = -\frac{c \cdot dP}{P + b} \quad \text{(const T)} \\ W &= -\int\limits_{V_1}^{V_2} P dV = -\int\limits_{V_1}^{V_2} P \cdot \bigg(-\frac{c}{P + b} \bigg) dP = \int\limits_{P_1}^{P_2} P \cdot \bigg(\frac{c}{P + b} \bigg) dP \end{split}$$

$$ln[*]:= b = 2700; (*bar*)$$

 $c = 0.125; (*\frac{cm^3}{g}*);$

(*Solve for absolute work*)

$$ln[*] = W = 1000 * \int_{1}^{500} \frac{c * P}{P + b} dP$$

Out[\bullet]= 5159.05193912

(*Units are
$$g * \frac{cm^3}{g} * bar * bar = bar * cm^3 *)$$

W * bar * cm³ *
$$\frac{1 \text{ J}}{10 \text{ bar * cm}^3}$$
 * $\frac{1 \text{ kJ}}{1000 \text{ J}}$ (*//ANS*)

Out = 0.515905193912 kJ

One mole of an ideal gas with $C_P = (7/2) R$ and $C_V = (5/2) R$ expands from $P_1 = 8$ bar and $T_1 = 600 K$ to $P_2 = 1$ bar by each of the following paths:

- (a) Constant volume,
- (b) Constant temperature,
- (c) Adiabatically

Assuming mechanical reversibility, calculate W, Q, ΔU , and ΔH for each process. Sketch each path on a single PV diagram.

Solution

```
R = 8.314; (*J/mol*K*)
CV = 20.785; (*J/mol*K*)
CP = 29.099; (*J/mol*K*)
T1 = 600; (*K*)
P1 = 8.; (*bar*)
P2 = 1.; (*bar*)
```

Part (a) - Constant volume

```
T2 = T1 * P2  (*K*)

75.

DU = CV * (T2 - T1)  (*J/mol*)

DH = CP * (T2 - T1)  (*J/mol*)

- 10912.125

- 15276.975

Q = DU (*DU=Q+W=Q+0*)

- 10912.125

W = 0;

(* W = 0 J/mol*)

(* Q = -10912.1 J/mol*)

(* \Delta U = -10912. J/mol*)

(* \Delta H = -15277. J/mol*)

(*/ANS part (a) *)
```

Part (b) - Constant temperature

$$DH = 0;$$

$$Q = -R * T1 * Log \left[\frac{P2}{P1} \right]$$

10373.0861865

$$W = -Q$$

-10373.0861865

$$(* W = -10373. J/mol*)$$

$$(* Q = 10373. J/mol*)$$

$$(*\Delta U = 0 \text{ J/mol*})$$

$$(*\Delta H = 0 J/mol*)$$

Part (c) - Adiabatic

$$Q = 0;$$

$$\gamma = CP / CV;$$

$$W = \frac{R \star T1}{\gamma - 1} \left(\left(\frac{P2}{P1} \right)^{(\gamma - 1)/\gamma} - 1 \right)$$

-5586.44983749

$$DU = W$$

-5586.44983749

$$T2 = T1 * \left(\frac{P2}{P1}\right)^{R/CP}$$

331.226854102

$$DH = CP * (T2 - T1)$$

-7821.02977248

$$(* W = -5586. J/mol*)$$

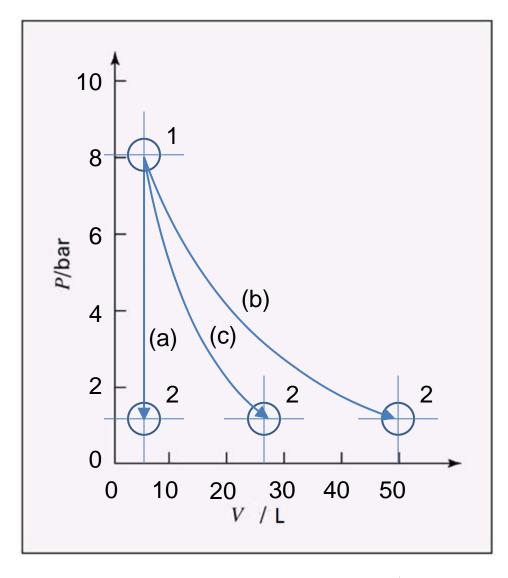
$$(* Q = 0. J/mol*)$$

$$(*\Delta U = -5586. \ J/mol*)$$

$$(*\Delta H = -7821. J/mol*)$$

Sketches - Parts (a) - (c)

(*Shown on following page*)



Givens:

 $P_1 = 8 \text{ bar}$

 $T_1 = 600 \text{ K}$

 $V_1 = 6.24 L$

 $P_2 = 1 \text{ bar}$

Part (a)

Constant volume

 $V_2 = 6.24 L$

Ideal Gas Equation gives conditions at state 2 for

parts (b) and (c)

Part (b)

Isothermal

 $P_2 = 1 bar$

 $T_2 = 600 \text{ K}$ $V_2 = 49.92$

 $P_1V_1 = P_2V_2$

Part (c)

Adiabatic

 $P_2 = 1 bar$

 $T_2^2 = 331.2 \text{ K}$

 $V_2 = 27.56 L$

$$T_2 = T_1 \left(\frac{P_2}{P_1}\right)^{R/C_p} = 600 \cdot \left(\frac{1}{8}\right)^{\frac{R}{7R/2}} = 331.2 \text{ K} \qquad V_2 = V_1 \left(\frac{P_2}{P_1}\right)^{\frac{-7/2}{5/2}} = 6.24 \cdot \left(\frac{1}{8}\right)^{\frac{-7/2}{5/2}} = 27.56 \text{ L}$$

An ideal gas, $C_P = (5/2) R$ and $C_V = (3/2) R$, is changed from $P_1 = 1$ bar and $V_1^t = 12 m^3$ to $P_2 = 12$ bar and $V_2^t = 1 m^3$ by the following mechanically reversible processes:

- (a) Isothermal compression.
- (b) Adiabatic compression followed by cooling at constant pressure.
- (c) Adiabatic compression followed by cooling at constant volume.
- (d) Heating at constant volume followed by cooling at constant pressure.
- (e) Cooling at constant pressure followed by heating at constant volume.

Calculate Q, W, ΔU^t , and ΔH^t for part (a) only. For parts (b) to (e), sketch the path of each process on a single PV diagram.

Solution

```
In[*]:= Quit[];
Common Information for Parts (a)-(e)

(*Since points 1 and 2 are defined on an isotherm in part (a) then ΔH=0 and ΔU=0.*)

(*This can be proven mathematically assuming a closed system:*)
(*Note that the total volume is given, not molar volume.*)
```

```
(*PV^{tot} = nRT*)
(*\frac{PV^{tot}}{n*T} = constant*)
(*Therefore, \frac{P_1V_1^{tot}}{n_1*T_1} = \frac{P_2V_2^{tot}}{n_2*T_2}*)
(*Assume n_1 = n_2 = n for a closed system so n cancels.*)
(*Since P_1V_1^{tot} = P_2V_2^{tot} (that is, 12*1=1*12) then T_1 = T_2 which means the overall process is isothermal.*)
```

```
Im[*]:= \Delta H = 0; (*//ANS, parts (a) - (e) *)
\Delta U = 0; (*//ANS, parts (a) - (e) *)
P1 = 1.; (*bar*)
P2 = 12.; (*bar*)
V1 = 12.; (*m^3*)
V2 = 1.; (*m^3*)
R = 8.314; (*J/mol*K*)
C_P = (5/2) * R;
C_V = (3/2) * R;
Y = C_P / C_V;
(*V, Q, H, and W are total (not molar)*)
```

Part (a) - Isothermal Compression

Part (b) - Adiabatic compression followed by cooling at constant pressure.

(*Let Vint be the intermediate volume.*)

eqb =
$$\frac{P2}{P1} = \left(\frac{V1}{Vint}\right)^{C_p/C_v}$$
; (*Un-numbered equation from L11 Slide 9*)

Vint = Vint /. Quiet[Solve[eqb, Vint]][1]

Outfel= 2.70192007704

(*STEP 1 - adiabatic compression*)

W1 =
$$\frac{P2 * Vint - P1 * V1}{\gamma - 1}$$
 (*bar*m³*) (*Eq 3.26 L11 slide 10 *)

 $Out[\bullet] = 30.6345613867$

(*STEP 2 - isobaric cooling at 12 bar*)

$$ln[@] := W2 = -P2 * (V2 - Vint) (*bar*m^3*)$$

Out = 20.4230409245

$$ln[\circ] := W = W1 + W2 (*bar*m^3*)$$

 $Out[\circ] = 51.0576023112$

$$lo[*] = W * bar * m^3 * \frac{1 J}{10^{-5} bar * m^3} * \frac{1 kJ}{1000 J} (*//ANS part (b) *)$$

 $Out[\circ] = 5105.76023112 \ kJ$

(*Since $\Delta U=0$, Q=-W=-5105.76 kJ //ANS (part(b)*)

Part (c) - Adiabatic compression followed by cooling at constant volume.

(*STEP 1 - adaibatic compression to 1
$$\mathrm{m}^3\star$$
)

(*Pint is the intermediate pressure*)

$$los_{los_{p}} = Pint = P1 * \left(\frac{V1}{V2}\right)^{C_{p}/C_{V}} (*un-numbered equation L11 slide*)$$

Out[*]= 62.897793461

$$ln[*]:=$$
 (*from slide 10 in lesson 12 eq 3.26 for work*)

$$W1 = \frac{Pint * V2 - P1 * V1}{\gamma - 1} (*bar*m^3*)$$

 $Out[\circ] = 76.3466901915$

(*STEP 2 - work is zero at const V*)

$$W = W1 + W2 \quad (*bar*m^3*)$$

 $Out[\circ] = 76.3466901915$

$$ln[a] = W * bar * m^3 * \frac{1J}{10^{-5} bar * m^3} * \frac{1 kJ}{1000 J} (*//ANS part (c) *)$$

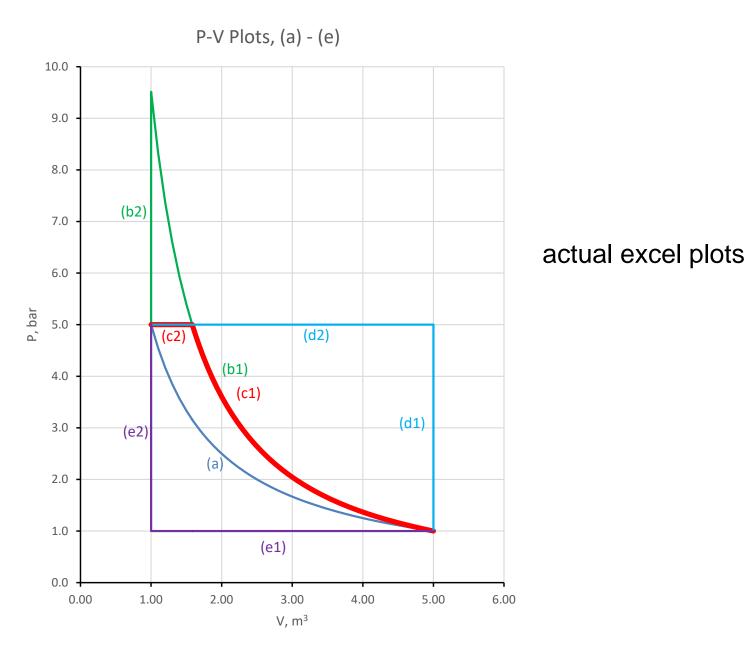
 $Out[\circ] = 7634.66901915 \text{ kJ}$

(*Since
$$\Delta U=0$$
, Q=-W=-7634.67 kJ //ANS part (c)*)

Part (d) - Heating at constant volume followed by cooling at constant pressure.

Part (e) - Cooling at constant pressure followed by heating at constant volume.

(*Since $\Delta U=0$, Q=-W=-13200 kJ //ANS part (d)*)



(a) isothermal compression

(b1) adiabatic compression

(c1) adiabatic compression

(c2) cooling at constant volume

(d1) heating at constant volume

(d2) cooling at constant pressure

(e1) cooling at constant pressure(e2) heating at constant volume

(b2) cooling at constant pressure

For submission, all five sketches must be on the same PV axes.

For methyl chloride at 100 degC, the second and third virial coefficients are:

$$B = -242.5 \,\mathrm{cm}^3 \,\mathrm{mol}^{-1}$$
 and $C = 25,200 \,\mathrm{cm}^6 \,\mathrm{mol}^{-2}$

Calculate the work of mechanically reversible, isothermal compression of 1 mole of methyl chloride from 1 bar to 55 bar at 100 degC. Base your calculations on the following forms of the virial equation:

(a)
$$Z = 1 + \frac{B}{V} + \frac{C}{V^2}$$

(b)
$$Z = 1 + B'P + C'P^2$$

(c)
$$Z = 1$$

(d)
$$Z = 1 + \frac{B}{V}$$

where
$$B' = \frac{B}{RT}$$
 and $C' = \frac{C - B^2}{(RT)^2}$

Why don't parts (a) and (b) give exactly the same result?

Solution

The key to this problem is understanding the reversible work equation (Equation 1.3) and then understanding how to integrate it. That is,

$$dW = -PdV$$

It is very important to realize that P=P(V), and that whether we are using the ideal gas law or the virial equation, we must substitute the function P(V) into equation 1.3 and integrate from V1 to V2. This means we must also determine V1 and V2.

What follows below for the solution to Part (a) is (1) a determination of V1 and V2, (2) determination of the function for P(V), then (3) integrating Equation 1.2 from V1 to V2.

Part (a)

Determination V1 and V2:

$$\label{eq:loss_problem} \begin{split} & \textit{In[s]} = \text{ eq1} = \frac{\text{P1} * \text{V1}}{\text{R} * \text{T}} = \text{1} + \frac{\text{B}}{\text{V1}} + \frac{\text{C}}{\text{V1}^2}; \\ & \textit{In[s]} = \text{Quiet[sol1} = \text{Solve[eq1, V1, Reals]]} \\ & \textit{Out[s]} = \left\{ \left\{ \text{V1} \to 30\,780.097043 \right\} \right\} \end{split}$$

V1 = V1 /. sol1[[1]] (*units are
$$\frac{cm^3}{mol}$$
*)

Outf =]= 30780.097043

$$log_{0} := eq2 = \frac{P2 * V2}{R * T} = 1 + \frac{B}{V2} + \frac{C}{V2^{2}};$$

In[*]:= Quiet[sol2 = Solve[eq2, V2, Reals]]

Out[*]= $\{ \{ V2 \rightarrow 241.333143561 \} \}$

$$ln[\circ]:= V2 = V2 /. sol2[1] (*units are $\frac{cm^3}{mol}*)$$$

 $Out[\circ] = 241.333143561$

Determination the function P(V):

$$ln[*] = P[V_] = \left(1 + \frac{B}{V} + \frac{C}{V^2}\right) * \frac{R * T}{V}; (*bar*)$$

Integrate equation 1.3:

$$log[a] := W = -\int_{V1}^{V2} (P[V]) dV (*\frac{bar*cm^3}{mol}*)$$

Out[*]= 126 198.643583

(*Put dimensions of work into kJ:*)

$$log[*] = 126199.* \frac{bar * cm^3}{mol} * \frac{10^5 Pa}{bar} * \frac{\frac{1}{m^2}}{Pa} * \left(\frac{1 m}{100 cm}\right)^3 * \frac{1 J}{1 N * m} * \frac{1 kJ}{1000 J} * 1 mol$$

Out[]= 12.6199 kJ

(*//ANS*)

Part (b)

The solution for Part (b) is a little different. In this case, we are asked to use the virial equation in the form of a power series expansion in P. In this equation (given above), V is easy to isolate, so the strategy is to isolate it, then evaluate dV. This produces an expression in terms of dP. Substitute the new dV into equation 1.3, and integrate with respect to pressure (instead of volume). This means the limits of integration ore P1 and P2, not V1 and V2 as in Part (a).

What follows below the solution to Part (b) is (1) a determination of V(P), then (2) determination work using Equation 1.3 after substituting in dV.

Determine the function V(P):

$$In[*] = Bprime = \frac{B}{R*T}; (*Eq 3-35a slide 13*)$$

$$Cprime = \frac{C-B^2}{(R*T)^2}; (*Eq 3-35b*)$$

$$(*Solve eq 3.34 for V in terms of P*)$$

$$In[*] = V[P_] = (1 + Bprime * P + Cprime * P^2) * \frac{R*T}{P};$$

Integrate equation 1.3 (dW=-PdV) with respect to P after substituting dV=V'dP:

$$In[*]:= W = -\int_{P1}^{P2} P * V'[P] dP$$

$$Out[*]:= 125960.132515$$

$$In[*]:= 125960. * * \frac{bar * cm^3}{mol} * * \frac{10^5 Pa}{bar} * * \frac{\frac{1N}{m^2}}{Pa} * (\frac{1m}{100 cm})^3 * * \frac{1J}{1N*m} * * \frac{1kJ}{1000 J} * 1 mol$$

$$Out[*]:= 12.596 kJ$$

Part (c)

(*//ANS*)

Proceed as in Part (a) - find V1 and V2, write P(V), substitute into equation 1.3, and integrate:

$$ln[\circ] := W = -\int_{V1}^{V2} Pideal[V] dV$$

Outf •]= 124 322.266473

$$ln[*] = 124322.* \frac{bar * cm^3}{mol} * \frac{10^5 Pa}{bar} * \frac{\frac{1N}{m^2}}{Pa} * \left(\frac{1m}{100 cm}\right)^3 * \frac{1J}{1N*m} * \frac{1kJ}{1000 J} * 1 mol$$

Out[*]= 12.4322 kJ

The work equation can also be integrated with respect to P/ as in part (b), resulting in exactly the same answer:

$$ln[*] := V[P_] = \frac{R * T}{P};$$

$$ln[\circ]:= W = -\int_{P1}^{P2} P * V'[P] dP$$

Out[*]= 124 322.266473

Part (d)

This problem is a little tricky because the equation is quadratic and there are no real roots for V2. Essentially, the problem cannot be solved as posed. We can come close, though. To get a reasonable approximation, use the values of V1 and V2 determined in Part (a).

$$ln[*] = P[V_] = \left(1 + \frac{B}{V}\right) * \frac{R * T}{V}; (*bar*)$$

$$ln[*]:= W = -\int_{V_1}^{V_2} P[V] dV$$

 $Out[\bullet] = 119487.395271$

$$lo[e] = 119487.* \frac{bar * cm^{3}}{mol} * \frac{10^{5} Pa}{bar} * \frac{\frac{1}{m^{2}}}{Pa} * \left(\frac{1 \text{ m}}{100 \text{ cm}}\right)^{3} * \frac{1 \text{ J}}{1 \text{ N} * m} * \frac{1 \text{ kJ}}{1000 \text{ J}} * 1 \text{ mol}$$

Out[*]= 11.9487 kJ

(*//ANS*)