CH402 Chemical Engineering Process Design

Class Notes L3

Pumps

BONUS OP

Chemical Engineering Plebe Open House

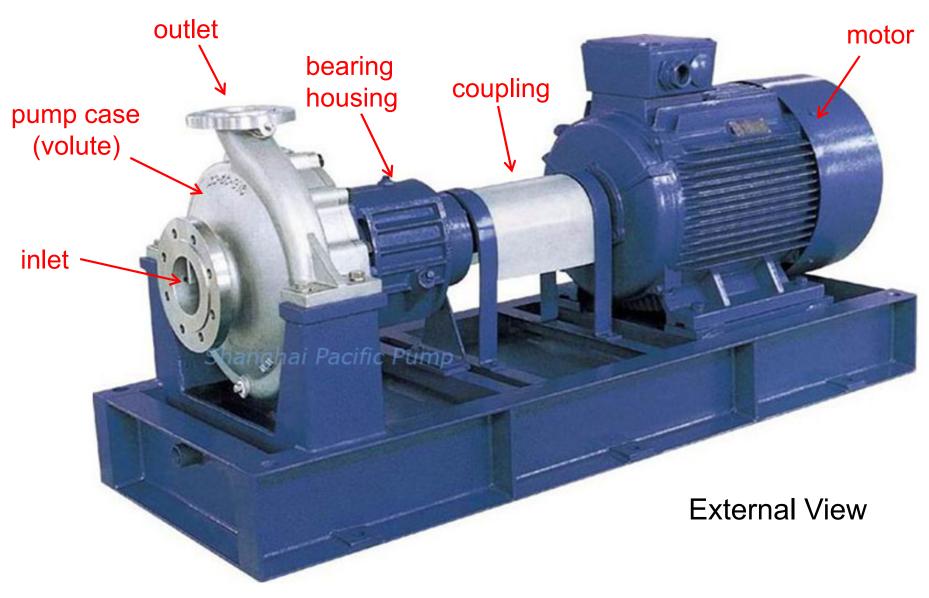
23 JAN 2025 1245 to ~1400 Bartlett Hall Room 150¹

 $30 \text{ minutes} = 5 \text{ points}^2$ Max 1.5 hours (15 points)

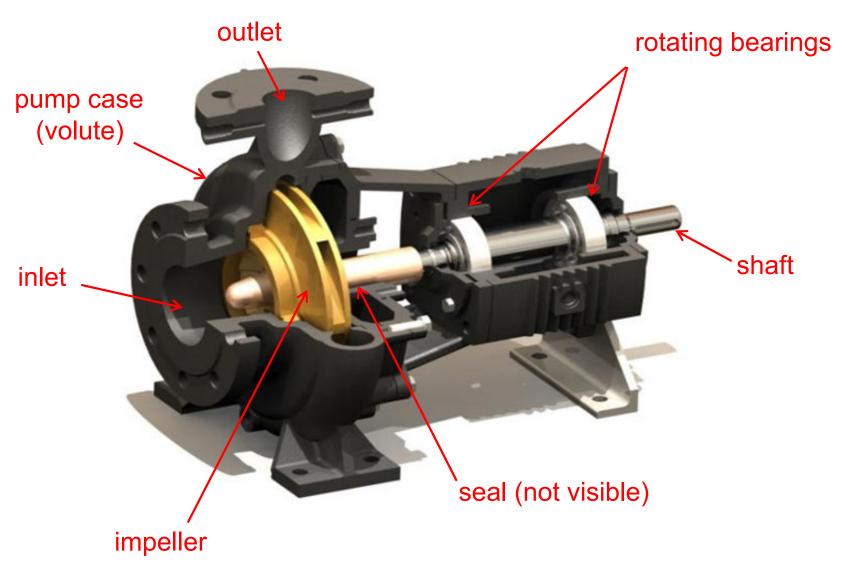
Notes:

- 1. If we are moved to a different location, we will still be somewhere near 150.
- 2. Sign in and out on the provided roster with time in and time out. Interact with prospective cadets. Stay active. Try not to congregate in friend clusters.

Pump Design Basics

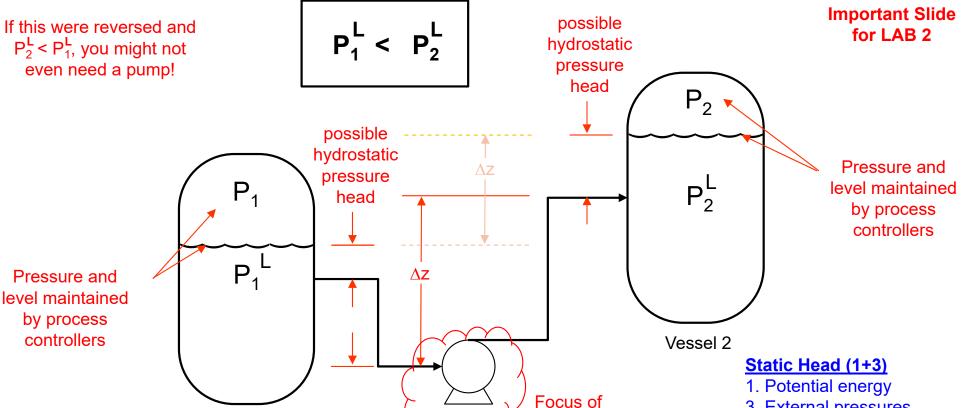


Pump Design Basics



Internal View

Pump Overview (Purpose)



• Pump power can be expressed power (W), pressure (Pa), or static pressure equivalent ("total head" or "meters of head")

$$W_o = g\Delta z + \Delta \left(\frac{V^2}{2\alpha}\right) + \frac{\Delta p}{\rho} + \sum F$$

Pump

attention in L3

• Pump power must be sufficient to overcome changes in elevation, kinetic energy changes, external pressure difference, and frictional losses.

Vessel 1

 Pump cost depends on pump power and pump flow rate, then materials & design details.

3. External pressures

(Static pump head is the sum of the potential energy and external pressure difference terms and does not depend on flow rate.)

Dynamic Head (2+4)

- 2. Frictional losses
- 4. Kinetic energy

Dynamic head is the sum of the kinetic energy and frictional loss terms and depends on the flow rate)

Pump Performance

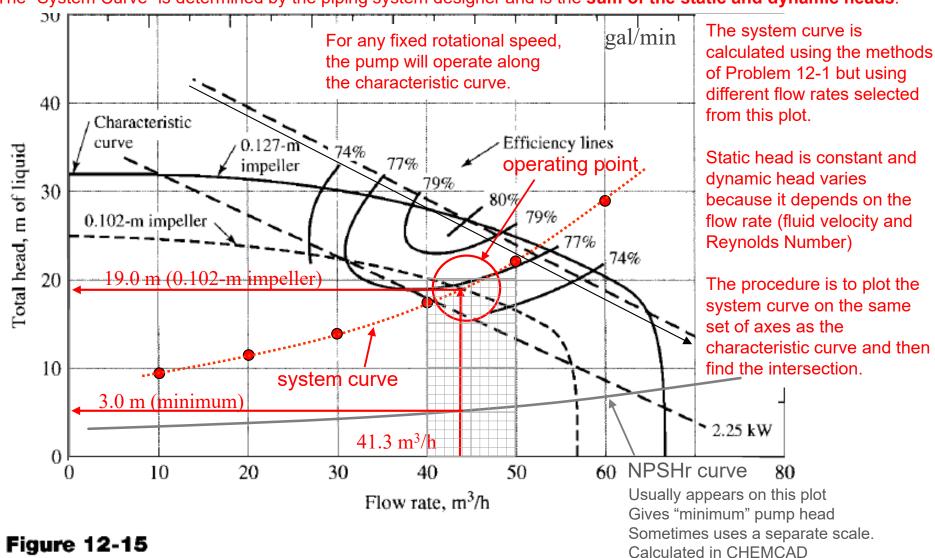
Important Slide for LAB 2

Depends primarily on the "Characteristic Curves"

The characteristic curve is determined by the manufacturer and is a plot of pump head versus flow rate.

Pump head is a combination of input power and friction, internal leakage and recirculation losses.

The "System Curve" is determined by the piping system designer and is the **sum of the static and dynamic heads**.



Characteristic curve of a centrifugal pump operating at a constant speed of 3450 r/min

NPSH (Net Positive Suction Head) is the "available" pressure present in the liquid at the entrance to the pump.

NPSHa Example Calculation

Conditions taken from feed stream in DP2

50/50 mol% mixture of toluene and ethylbenzene

T = 20 deg C (293 K)P = 101325 Pa

In[10]:=
$$Pa * \frac{\frac{N}{m^2}}{Pa} * \frac{\frac{kg*m}{s^2}}{N}$$

Out[10]= $\frac{kg}{m s^2}$

$$(*\Delta P = \rho * g * h \rightarrow h = \Delta P / \rho * g *)$$

In[11]:= $\frac{Pa * \frac{\frac{N}{m^2}}{Pa} * \frac{\frac{kg*m}{s^2}}{N}}{\frac{kg}{m^3} * \frac{m}{s^2}}$

Out[11]= m

101 325 - P[293.] 868.1519 * 9.8

"available" suction head

NPSHa

In[12]:=

Out[12]= **11.6834**

Design Procedure for Pumps

Mechanical Energy Balance – Equation 12-12 – Excel, Mathematica, CAD, By-hand (same as we used in problem set 1)

Total pressure developed by pump

$$\frac{\mathbf{m}}{\mathbf{s}^2} \cdot \mathbf{m} = \frac{\mathbf{m}^2}{\mathbf{s}^2}$$

$$\frac{m^2}{s^2} \cdot \frac{kg}{s} = \frac{kg \cdot m^2}{s^3} = Watts$$

Total pressure developed by pump
$$\frac{m}{s^2} \cdot m = \frac{m^2}{s^2} \qquad \frac{m^2}{s^2} \cdot \frac{kg}{s} = \frac{kg \cdot m^2}{s^3} = \text{Watts} \qquad w_0 = \frac{\Delta p}{\rho} = g \cdot H \text{ specific work; from eq 12-12}$$

pump work equations on page 515

in eq 12-20a, H in units of Nm/kg

$$\frac{\mathbf{N} \cdot \mathbf{m}}{\mathbf{kg}} = \frac{\mathbf{kg} \cdot \mathbf{m}}{\mathbf{s}^2} \cdot \frac{\mathbf{m}}{\mathbf{kg}} = \frac{\mathbf{m}^2}{\mathbf{s}^2}$$

In eq 12-20b,

H in units of N/m²

$$\mathbf{w}_0 = \frac{\mathbf{H} \cdot \dot{\mathbf{m}}_{\mathbf{v}} \cdot \mathbf{p}}{10^3}$$

$$\mathbf{w}_0 = \frac{\mathbf{H} \cdot \dot{\mathbf{m}}_{\mathbf{v}}}{10^3}$$

$$\mathbf{w}_{0} = \frac{\mathbf{H} \cdot \dot{\mathbf{m}}_{v} \cdot \boldsymbol{\rho}}{10^{3}} \qquad \underbrace{\frac{N \cdot \mathbf{m}}{kg} \cdot \frac{m^{3}}{s} \cdot \frac{kg}{m^{3}}}_{1N = 1} = \underbrace{\frac{kg \cdot \mathbf{m}}{s^{2}} \cdot \frac{m}{kg}}_{1N = 1} \cdot \underbrace{\frac{kg \cdot \mathbf{m}}{s^{2}} \cdot \frac{kg}{m^{3}}}_{1N = 1} = \underbrace{\frac{kg \cdot \mathbf{m}}{s^{2}}}_{1N = 1} \cdot \underbrace{\frac{kg \cdot \mathbf{m}}{s^{2}}}_{1N = 1} = \underbrace{\frac{k$$

$$\mathbf{W}_0 = \frac{\mathbf{H} \cdot \mathbf{m}_{v}}{10^3} \qquad \frac{\mathbf{N}}{\mathbf{m}^2} \cdot \frac{\mathbf{m}^3}{\mathbf{s}} = \frac{\mathbf{N} \cdot \mathbf{m}}{\mathbf{s}} = \frac{\mathbf{J}}{\mathbf{s}} = \text{Watts}$$

Cavitation

$$NPSH = \frac{1}{g} \cdot \left(\frac{p_{reference} - p_{vapor}}{\rho} - h_f \right) - Z_{ref}$$
 Typically 2-5 m for small pumps And up to 15 m for large pumps See McCabe, et al, page 204

Frictional losses

Efficiency

$$\eta = \frac{W_0}{W}$$

Use Fig. 12-17, page 516

Cost

Use Figs 12-19 through 12-24, pages 517-520; PTW website; CHEMCAD

Pump Video Links - Watch

multistage (1st two minutes)

centrifugal explained

pump internals

cavitation sound

simple piston

pump curve
(Jacques Chaurette, #2)

cavitation

cavitation explained

centrifugal force (fantastic demo)

Questions

Homework

PROBLEM SET 2

Problem 12-6

A preliminary estimate of the total cost for a completely installed pumping system is required for a certain design project. In this system, 15.75 kg/s of cooling water at 15.5 °C is to be provided using 305-m pipeline. It has been estimated that the theoretical power requirements for the pump will be 7.5 kW. Using the following data, estimate the total cost of the pumping system:

Material of construction – carbon steel Insulation (85% magnesia) – 0.038 m

Number of fittings (equivalent to tees) – 40 Pump – centrifugal

Number of valves (gate) – 4 Motor – AC, enclosed, 3-phase, 1800 r/min

COST MUST BE PURCHASED INSTALLED COST IN JANUARY 2025

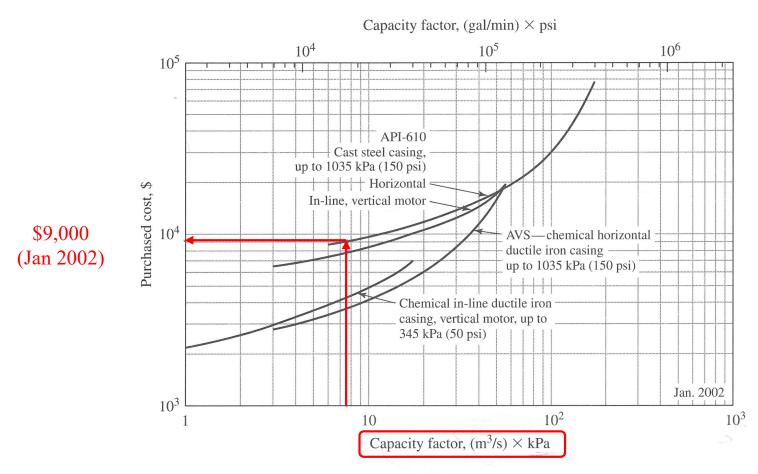


Figure 12-20Purchased cost of centrifugal pumps. Price includes electric motor.