# CH402 Chemical Engineering Process Design

Class Notes L8

Heat Exchanger Types and Costs

# Today's agenda

Finish problem 14-2

General design steps.

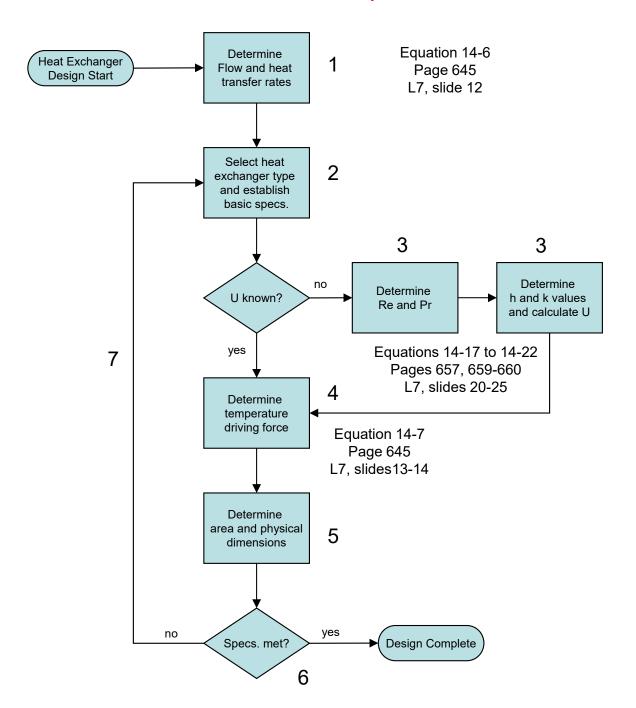
General features of double-pipe and shell-and-tube exchangers.

Cost correlations

Problem 14-9

#### Steps in Heat Exchanger Design

#### Implemented in the "3-step method"

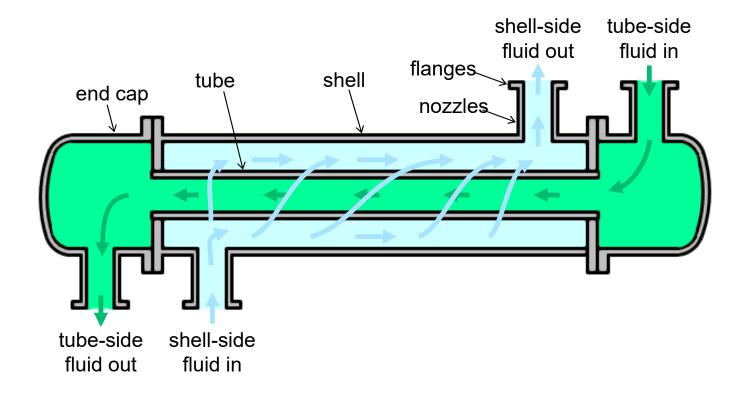


- 1. Determine the flow rates and heat transfer rates necessary to meet the given conditions.
- 2. Select the type of heat exchanger to be used and establish basic specifications.
- 3. Evaluate the overall heat transfer coefficient.
- 4. Evaluate the temperature driving force.
- 5. Determine the required heat transfer area.
- Analyze dimensions, pressure drops, capital and operating costs.
- 7. If Step 6 reveals unsatisfactory performance, go to Step 2 and repeat.

## Types of Heat Exchangers

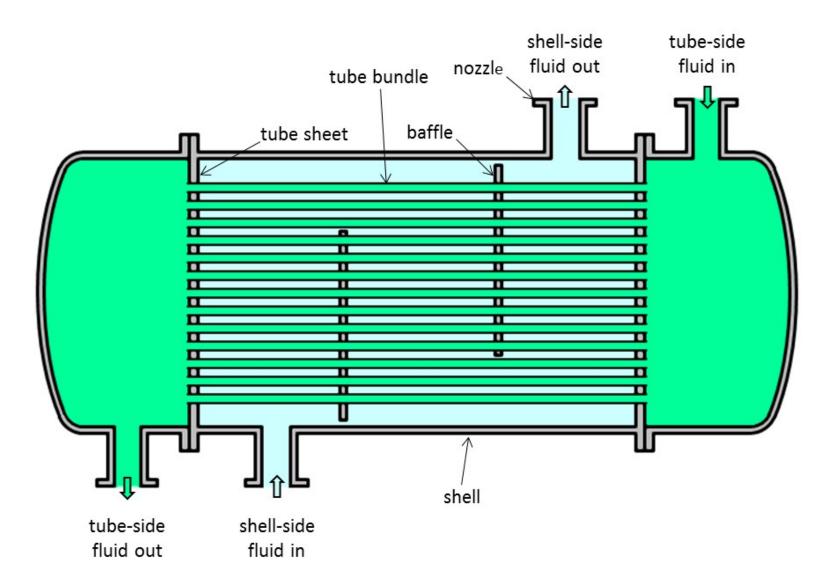
- Double-pipe
- Shell-and-tube
- Reboilers
- Scraped-surface
- Welded-plate

- Spiral
- Compact
- Air-cooled
- Evaporators



Simplest design – "tube inside a tube." Highly modular (U-tubes).

Works well when heat transfer rates are small. Fins may be needed (inside or outside tube). Easy to clean.



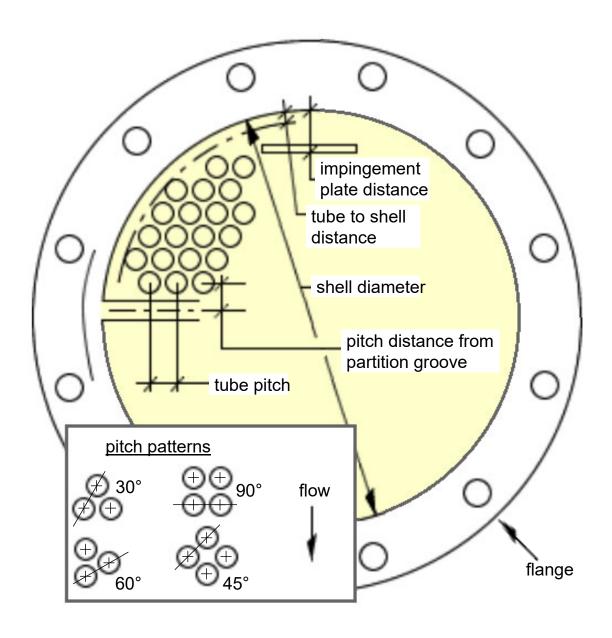
Simplest design – "tube inside a tube."

Highly modular (U-tubes).

Works well when:

hath incide and autoide at high P

#### Shell and tube – tube sheet layout





## **Tubular Exchanger Manufacturers** Association, Inc.

Home

Members

TEMA Advantage

Standards & Software

Contact

#### CHEMCAD design provides "TEMA Sheets"

The Tubular Exchanger Manufacturers Association, Inc. (TEMA) is trade association of leading manufacturers of shell and tube heat exchangers, who have pioneered the research and development of heat exchangers for over sixty years.

The TEMA Standards and software have achieved worldwide acceptance as the authority on shell and tube heat exchanger mechanical design.

TEMA is a progressive organization with an eye towards the future. Members are market-aware and actively involved, meeting several times a year to discuss current trends in design and manufacturing. The internal organization includes various subdivisions committed to solving technical problems and improving equipment performance. This cooperative technical effort creates an extensive network for problem-solving, adding value from design to fabrication.

Whether having a heat exchanger designed, fabricated or repaired, you can count on TEMA members to provide the most current, efficient design and manufacturing solutions. TEMA is a way of thinking--members are not only researching the latest technology, they're creating it.

For over half a century our main goal has been to continually find innovated TEMA Plate attached to the heat exchanger. approaches to heat exchanger applications. As a result, TEMA members have a When you deal with a TEMA manufacturer, you unique ability to understand and anticipate the technical and practical needs of enter into a partnership with an organization today's market.

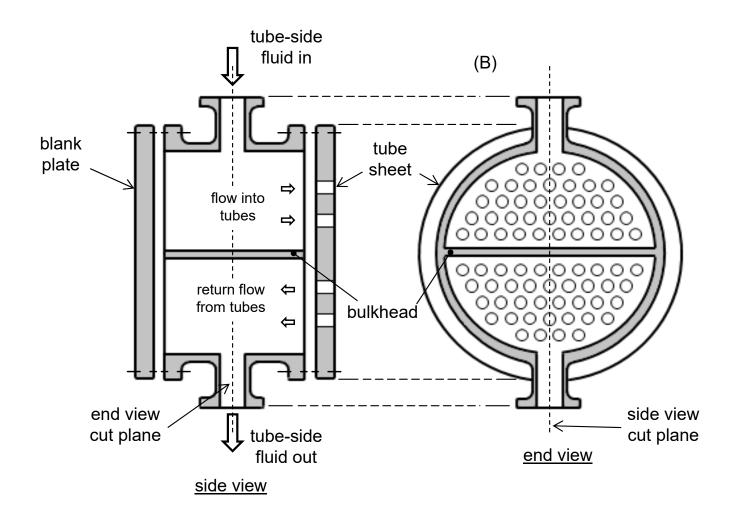
Using TEMA members as a resource today ensures a reliable partners for years to come.

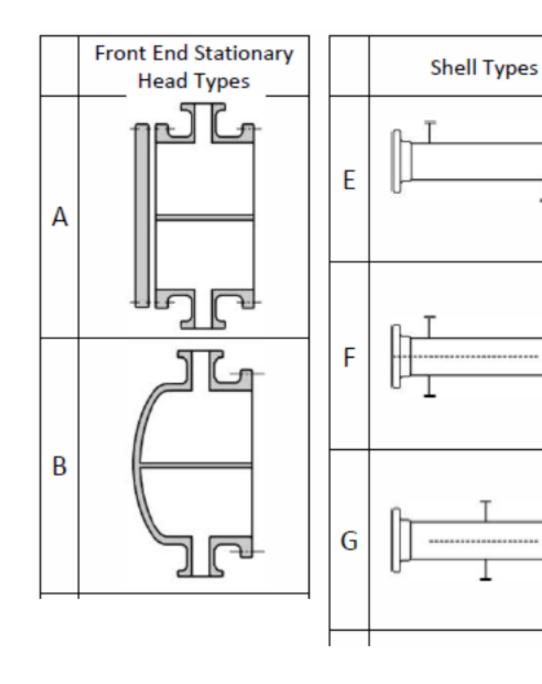


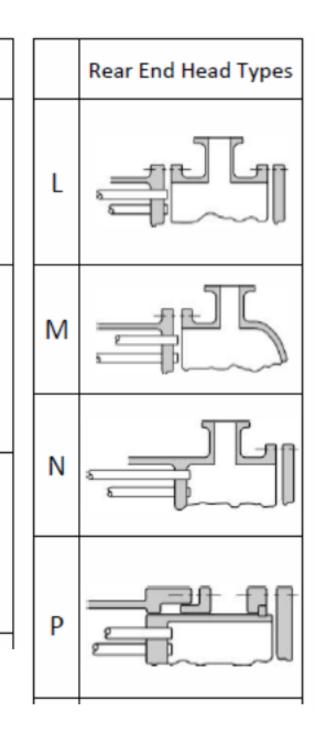
**TEMA Name Plate** 

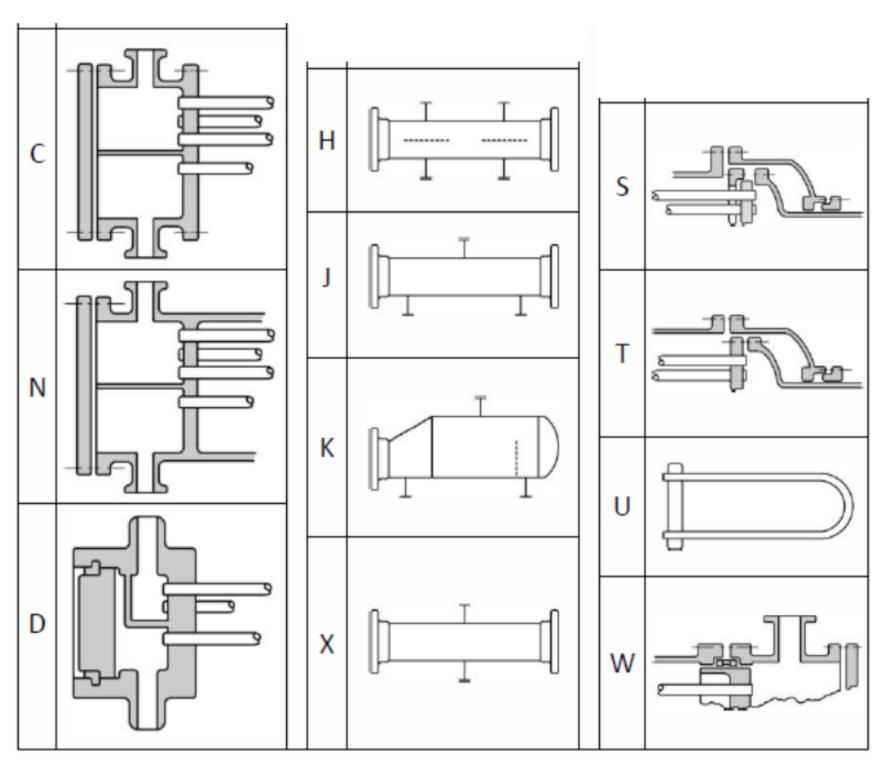
For quality assurance, one need only look for the dedicated to furnishing a product of the highest technical standards.

#### Shell-and-tube – TEMA type A front end details.









#### **Selection Criteria**

| Туре           | Max. P, MPa              | T, °C        | Area, m <sup>2</sup> | velocity,<br>shell/tube, m/s      | fluid limitations                              | key features                           |
|----------------|--------------------------|--------------|----------------------|-----------------------------------|--|--|
| Double-Pipe    | 30 (shell)<br>140 (tube) | -100 to 600  | .25-20               | liq., 2-3/2-3<br>gas, 10-20/10-20 | materials of<br>construction                   | modular, small<br>scale                |
| Multiple Pipe  | same                     | same         | 10-200               | same                              | same   | same                                   |
| Shell-and-Tube | same                     | -200 to 600+ | 3-1000               | liq., 1-3/2-3<br>gas, 5-10/10-20  | same   | adaptable                              |
| Scraped-wall   | ~0.11                    | up to 200    | 2-20                 | liq., 1-2/1-2                     | liquids<br>solidifying                         | for viscous,<br>crystallization        |
| Gasketed Plate | 0.1-2.5                  | -25-175      | 1-2500               | liq., 1-2/1-2<br>gas, 5-10/5-10   | gasket material;<br>avoid gases                | modular,<br>minimal \$/m <sup>2</sup>  |
| Welded Plate   | 3                        | >400         | 1-2500               | liq., 1-2/1-2<br>gas, 5-10/5-10   | materials of<br>construction;<br>fouling       | $\Delta p$ between fluids $\leq 3$ MPa |
| Spiral Plate   | 2                        | up to 300    | 10-200               | liq., 1-2/1-2<br>gas, 5-10/5-10   | materials of construction                      | viscous,<br>corrosive liq.             |
| Spiral Tube    | 50                       | 350          | 1-50                 | liq., 2-3/2-3<br>gas, 5-10/5-10   | materials of construction                      | adaptable                              |
| Compact        | 3-10                     | -270 to 800  | 10-30,000            | gas, 2-5/2-5                      | materials of<br>construction; no<br>corrosives | large area/vol;<br>very small ΔT       |

Table 14-6, page 677 and Table 14-7, page 678.

## Cost correlations

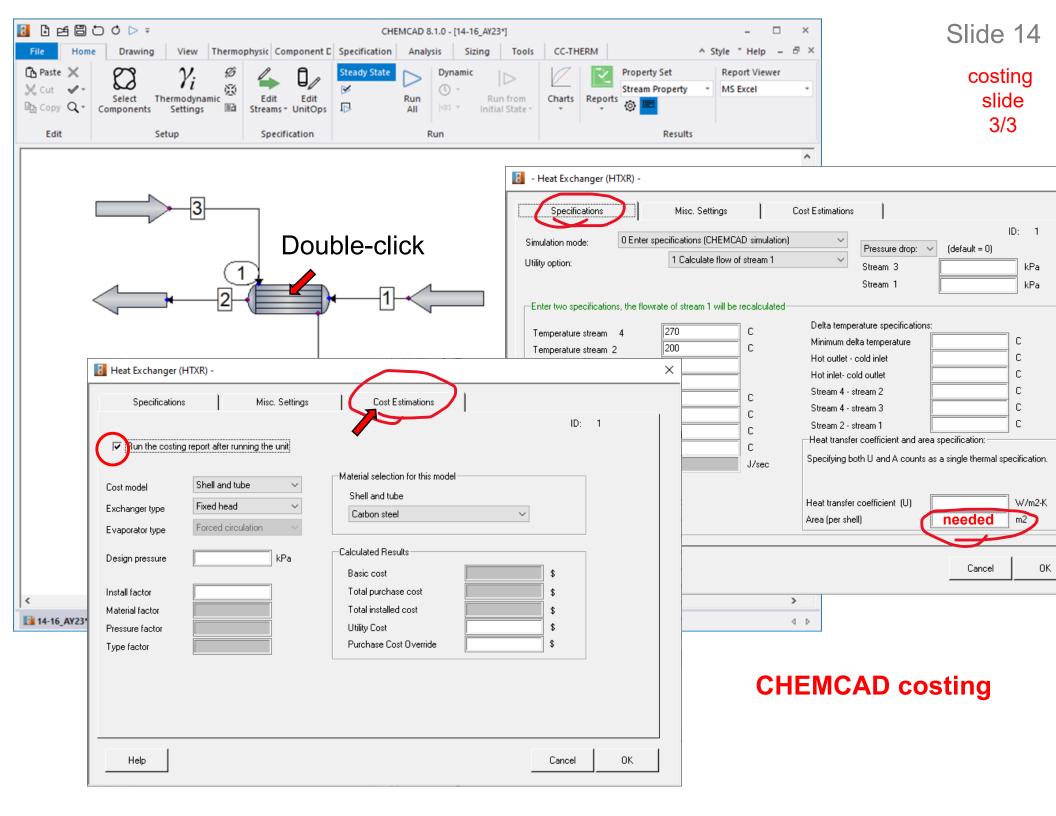
Figures and Tables from PTW

### Textbook figures

| Pricing of:               | <u>Table</u>           | <u>page</u> |  |
|---------------------------|------------------------|-------------|--|
| Double-pipe               |                        |             |  |
| Double-pipe               | 14-15                  | 680         |  |
| Multiple double-pipe      | 14-16                  | 681         |  |
| Shell-and-tube            |                        |             |  |
| U-tube                    | 14-17                  | 681         |  |
| Fixed                     | 14-18                  | 682         |  |
| Floating                  | 14-19                  | 682         |  |
| Finned-tube floating      | 14-20                  | 683         |  |
| Effect of tube diameter   | 14-21                  | 683         |  |
| Effect of tube length     | 14-22                  | 684         |  |
| Effect of pressure        | 14-23                  | 684         |  |
| Effect of materials       | Table 14-8, Fig. 14-24 | 685         |  |
| Gasketed and welded plate | 14-25                  | 686         |  |
| Scraped wall and spiral   | 14-26                  | 686         |  |
| Spiral and flat plate     | 14-27                  | 687         |  |
| Air-cooled                | 14-28                  | 687         |  |
| Condensers                | 14-29 to 14-31         | 688-89      |  |
| Evaporators               | 14-32 to 14-34         | 689-90      |  |

- Online tools
- CHEMCAD

costing slide 1/3



## Homework

#### **Problem 14-2 – Part 1 – Local heat transfer coefficients (FEE)**

At an average film temperature of 350 K, what are the individual heat transfer coefficients when the fluid flowing in a 0.0254-m inside diameter tube is air, water or oil? Each fluid in this comparison exhibits a Reynolds number of  $5\times10^4$ . How would the pressure drop vary for each fluid? The relevant properties of the three fluids at 350 K are listed in the table below.

|                             | Air                | Water                 | Oil                   |
|-----------------------------|--------------------|-----------------------|-----------------------|
| Density, kg/m <sup>3</sup>  | .955               | 973                   | 854                   |
| Viscosity, Pa·s             | 2×10 <sup>-5</sup> | 3.72×10 <sup>-4</sup> | 3.56×10 <sup>-2</sup> |
| Thermal Conductivity, W/m·K | 0.030              | 0.668                 | 0.138                 |
| Heat Capacity, J/kg·K       | 1050               | 4190                  | 2116                  |

#### **Problem 14-2 – Part 2 – Pressure Drops (FEE)**

At an average film temperature of 350 K, what are the individual heat transfer coefficients when the fluid flowing in a 0.0254-m inside diameter tube is air, water or oil? Each fluid in this comparison exhibits a Reynolds number of  $5\times10^4$ . How would the pressure drop vary for each fluid? The relevant properties of the three fluids at 350 K are listed in the table below.

|                             | Air                | Water                 | Oil                   |
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| Density, kg/m <sup>3</sup>  | .955               | 973                   | 854                   |
| Viscosity, Pa·s             | 2×10 <sup>-5</sup> | 3.72×10 <sup>-4</sup> | 3.56×10 <sup>-2</sup> |
| Thermal Conductivity, W/m·K | 0.030              | 0.668                 | 0.138                 |
| Heat Capacity, J/kg·K       | 1050               | 4190                  | 2116                  |

#### Problem 14-9 (FEE)

A heat exchanger is to be constructed by forming copper tubing into a coil and placing the coil inside an insulated steel shell. In this exchanger, water will flow inside the tubing, and a hydrocarbon vapor will be condensing on the outside surface of the tubing at a rate of 0.126 kg/s. The tubing has an inside diameter of 0.0127 m and an outside diameter of 0.0152 m. The inlet temperature is 10 °C and the exit temperature is 32 °C. The heat of condensation of the hydrocarbon at the condensation temperature of 88 °C is 335 kJ/kg. The heat transfer coefficient for the condensing vapor is 1420 W/m<sup>2</sup>·K. Heat losses from the shell may be neglected. What length of copper tubing will be required to accomplish the desired heat transfer?

## Questions?