## Linearization problem

Consider a non-linear time-invariant system of the form

$$\begin{cases} \dot{x}(t) = f(x(t), u(t)), \\ y(t) = g(x(t), u(t)) \end{cases}$$
 (1)

such that  $x \in \mathbb{R}^n$ ,  $u \in \mathbb{R}^m$ ,  $y \in \mathbb{R}^p$ , and

A:  $\frac{\partial f(x,u)}{\partial x}$ ,  $\frac{\partial f(x,u)}{\partial u}$ ,  $\frac{\partial g(x,u)}{\partial x}$ , and  $\frac{\partial g(x,u)}{\partial u}$  exist and are continuous.

The question is:

X How to approximate system (1) by a linear model?

#### The simplest example

Consider the static time-invariant system

$$y(t) = ku(t) + \eta. \tag{2}$$

This system is nonlinear, but intuitively it seems to be "almost linear." Indeed, this system can easily be made linear. To this end notice that for any constant  $\mathfrak{u}(t)=\mathfrak{u}_e$  the output  $\mathfrak{y}(t)$  is also constant and given by  $\mathfrak{y}_e \doteq k\mathfrak{u}_e + \eta.$  Let's call the pair  $(\mathfrak{y}_e,\mathfrak{u}_e)$  the equilibrium point.

Define now new signals:

$$y_{\delta}(t) \doteq y(t) - y_{e} \quad \text{ and } \quad u_{\delta}(t) \doteq u(t) - u_{e},$$

which are deviations of y and  $\mathfrak u$  from the equilibrium point. Then

$$y_{\delta}(t) = ku_{\delta}(t),$$
 (2<sub>\delta</sub>)

which is *linear*. Thus, nonlinear system (2) can be made linear by an appropriate (cf. equilibrium) shift of its input and output signals.

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Linearization

## A little bit more complicated example

Consider now the system

$$\begin{cases} \dot{x}(t) = -ax(t) + bu(t) + \eta_x, \\ y(t) = x(t) + \eta_y. \end{cases}$$
 (3)

Assume again that the input  $\mathfrak{u}(t)=\mathfrak{u}_{e}$  is constant. Then

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$$x(t) = \int_0^t e^{-\alpha(t-s)}(bu_e + \eta_x)dt = \tfrac{1}{\alpha}(1-e^{-\alpha t})(bu_e + \eta_x).$$

As  $\lim_{t\to\infty}e^{-\alpha t}=$  0, system (3) approaches it equilibrium point  $^{\rm a}$   $(x_e,u_e)$  , where

$$x_e = \frac{1}{a}(bu_e + \eta_x).$$

#### A little bit more complicated example (contd)

Let's write now system (3) in terms of the deviations

$$x_{\delta}(t) \doteq x(t) - x_{e}$$
 and  $u_{\delta}(t) \doteq u(t) - u_{e}$ .

We get:

$$\begin{cases} \dot{x}_{\delta}(t) = -ax_{\delta}(t) + bu_{\delta}(t), \\ y_{\delta}(t) = x_{\delta}(t), \end{cases} \tag{3}_{\delta}$$

where  $y_{\delta}(t) \doteq y(t) - x_{\varepsilon} - \eta_{y}$ . Equation (3<sub>\delta</sub>) is thus a linear equivalent of the nonlinear system (3).

<sup>&</sup>lt;sup>a</sup>This is called the *steady-state*.

#### **Equilibrium**

The pair  $(x_e, u_e)$  is called *equilibrium* if

$$f(x_e, u_e) = 0.$$

Let  $(x_e, u_e)$  be an equilibrium of (1). Then x(t), y(t), and u(t) can be presented as follows:

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$$x(t) = x_e + x_\delta(t), \quad y(t) = y_e + y_\delta(t), \quad \text{and} \quad u(t) = u_e + u_\delta(t),$$

where  $y_e \doteq g(x_e, u_e)$ .

Eqn. (1) can then be rewritten as follows:

$$\begin{cases} \dot{x}_{\delta}(t) = f(x_e + x_{\delta}(t), u_e + u_{\delta}(t)), \\ y_{\delta}(t) = g(x_e + x_{\delta}(t), u_e + u_{\delta}(t)) - y_e \end{cases} \tag{1}_{\delta}$$

#### Tailor series expansion

One can expand  $f(\cdot)$  and  $g(\cdot)$  as follows<sup>a</sup>:

$$\begin{split} f\big(x,u\big) &= f(x_e,u_e) + \left. \tfrac{\partial f}{\partial x} \right|_{x_e,u_e} x_\delta + \left. \tfrac{\partial f}{\partial u} \right|_{x_e,u_e} u_\delta + o\big(x_\delta,u_\delta\big), \\ g\big(x,u\big) &= g(x_e,u_e) + \left. \tfrac{\partial g}{\partial x} \right|_{x_e,u_e} x_\delta + \left. \tfrac{\partial g}{\partial u} \right|_{x_e,u_e} u_\delta + o\big(x_\delta,u_\delta\big), \end{split}$$

where  $o(\eta)$  is any function of a variable  $\eta$  such that  $\lim_{\|\eta\|\to 0} \frac{\|o(\eta)\|}{\|\eta\|} = 0$ .

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$$\frac{\frac{\partial \varphi(\eta)}{\partial \eta} \doteq \left[ \begin{array}{ccc} \frac{\partial \varphi_1(\eta)}{\partial \eta_1} & \cdots & \frac{\partial \varphi_1(\eta)}{\partial \eta_m} \\ \vdots & & \vdots \\ \frac{\partial \varphi_n(\eta)}{\partial \eta_1} & \cdots & \frac{\partial \varphi_n(\eta)}{\partial \eta_m} \end{array} \right].$$

 $<sup>^</sup>a$  Given an n-dimensional function  $\varphi(\eta)$  of an m-dimensional argument  $\eta,$  the partial derivative of  $\varphi$  with respect to  $\eta$  (the <code>Jacobian matrix</code>) is defined as the following  $n\times m$  matrix:

# Linearization around $(x_e, u_e)$

If the deviations  $x_{\delta}$  and  $u_{\delta}$  are "small enough," then Eqn.  $(1_{\delta})$  can be approximated by the following *linear* model:

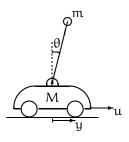
$$\begin{cases} \dot{x}_{\delta}(t) = A_{\delta}x_{\delta}(t) + B_{\delta}u_{\delta}(t), \\ y_{\delta}(t) = C_{\delta}x_{\delta}(t) + D_{\delta}u_{\delta}(t), \end{cases} \tag{1}_{\text{linearized}}$$

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where

$$\begin{split} A_{\delta} &\doteq \frac{\partial f}{\partial x}\big|_{x_{e}, u_{e}}, \\ C_{\delta} &\doteq \frac{\partial g}{\partial x}\big|_{x_{e}, u_{e}}, \\ D_{\delta} &\doteq \frac{\partial g}{\partial u}\big|_{x_{e}, u_{e}}. \end{split}$$

## **Inverted pendulum**



M: mass of the cart

 $\mathfrak{m}$ : mass of the pendulum

l : length of the pendulum

y : position of the cart

 $\boldsymbol{\theta}$  : angular rotation

 $\boldsymbol{\mathfrak{u}}\,$  : force on the cart

 $\boldsymbol{\mu}$  : friction coefficient

 $g\,$ : acceleration of gravity

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Nonlinear model:

$$\label{eq:linear_equation} \left\{ \begin{split} (M+m)\ddot{y} + ml\,\ddot{\theta}\cos\theta - ml(\dot{\theta})^2\sin\theta + \mu\dot{y} &= u\\ l\ddot{\theta} - g\sin\theta + \ddot{y}\cos\theta &= 0 \end{split} \right.$$

or, equivalently,

$$\left[ \begin{array}{cc} M+\mathfrak{m} & \mathfrak{m} l \cos \theta \\ \cos \theta & l \end{array} \right] \left[ \begin{array}{c} \ddot{y} \\ \ddot{\theta} \end{array} \right] = \left[ \begin{array}{c} \mathfrak{m} l (\dot{\theta})^2 \sin \theta - \mu \dot{y} \\ g \sin \theta \end{array} \right] + \left[ \begin{array}{c} 1 \\ 0 \end{array} \right] \mathfrak{u}.$$

## Inverted pendulum: the state-space realization

The state vector:

$$\mathbf{x} = \begin{bmatrix} \mathbf{x}_1 \\ \mathbf{x}_2 \\ \mathbf{x}_3 \\ \mathbf{x}_4 \end{bmatrix} = \begin{bmatrix} \mathbf{y} \\ \mathbf{\theta} \\ \dot{\mathbf{y}} \\ \dot{\mathbf{\theta}} \end{bmatrix}$$

Slide 9 The state equation:

$$\frac{d}{dt} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} = \underbrace{\begin{bmatrix} x_3 \\ x_4 \\ \left[ M+m \ ml\cos x_2 \\ \cos x_2 \ l \end{bmatrix}^{-1} \left( \begin{bmatrix} mlx_4^2 \sin x_2 - \mu x_3 \\ g\sin x_2 \end{bmatrix} + \begin{bmatrix} 1 \\ 0 \end{bmatrix} u \right)}_{f(x,u)}. \tag{4}$$

 $\text{ where } \Lambda(x_2) \doteq \left[ \begin{smallmatrix} M+\mathfrak{m} & \mathfrak{m} l\cos x_2 \\ \cos x_2 & l \end{smallmatrix} \right] \text{ is nonsingular } (\det \Lambda(x_2) = l(M+\mathfrak{m} \sin^2 x_2)).$ 

#### Inverted pendulum: equilibriums

Equilibrium points should correspond to  $\dot{x}\equiv 0$ . Then, from Eq. (4) one can get that  $x_3\equiv 0,\ x_4\equiv 0,$  and

$$\Lambda(x_2)^{-1} \left[ \begin{array}{c} u \\ g \sin x_2 \end{array} \right] = 0.$$

Therefore, the equilibrium points are:

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$$(x_e,u_e)=\left(\begin{bmatrix}x_1\\k\pi\\0\\0\end{bmatrix},0\right), \qquad \text{for every } x_1\in\mathbb{R} \text{ and } k=0,\pm 1,\pm 2,\dots$$

## Inverted pendulum: Jacobian matrices

Jacobian matricesa:

$$\begin{split} &\frac{\partial f_{1,2}}{\partial x} = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}, \\ &\frac{\partial f_{3,4}}{\partial x} = \Lambda(x_2)^{-1} \begin{bmatrix} 0 & mlx_4^2 \cos x_2 + \varphi_1(x_2, x_3, x_4) \sin x_2 & -\mu & 2mlx_4 \sin x_2 \\ 0 & g \cos x_2 + \varphi_2(x_2, x_3, x_4) \sin x_2 & 0 & 0 \end{bmatrix}, \end{split}$$

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where  $\phi_1$  and  $\phi_2$  are some algebraic functions of  $x_2$ ,  $x_3$ , and  $x_4$ ,

$$\frac{\partial f}{\partial u} = \frac{1}{l(M + m \sin^2 x_2)} \begin{bmatrix} 0 \\ 0 \\ l \\ -\cos x_2 \end{bmatrix}.$$

<sup>a</sup>Given two matrix functions  $M_1(\eta)$  and  $M_2(\eta)$ , then

$$\tfrac{\vartheta}{\vartheta\eta}\big(M_1(\eta)^{-1}M_2(\eta)\big)=M_1(\eta)^{-1}\big(\tfrac{\vartheta M_2(\eta)}{\vartheta\eta}-\tfrac{\vartheta M_1(\eta)}{\vartheta\eta}M_1(\eta)^{-1}M_2(\eta)\big).$$

## Inverted pendulum: linearization around (0,0)

It's readily verified that

$$\frac{\partial f}{\partial x} \Big|_{0} = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & -\frac{m}{M}g & -\frac{\mu}{M} & 0 \\ 0 & \frac{m+M}{Ml}g & \frac{\mu}{Ml} & 0 \end{bmatrix}$$

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and

$$\frac{\partial f}{\partial u}\big|_0 = \begin{bmatrix} 0 \\ 0 \\ \frac{1}{M} \\ -\frac{1}{MI} \end{bmatrix},$$

which yield the "A" and "B" matrices of the linearized realization.

# Linearization of discrete-time systems

The linearization problem for discrete-time system

$$\left\{ \begin{aligned} x_{k+1} &= f\big(x_k, u_k\big), \\ y_k &= g\big(x_k, u_k\big) \end{aligned} \right.$$

where  $x\in\mathbb{R}^n$ ,  $u\in\mathbb{R}^m$ ,  $y\in\mathbb{R}^p$ , has exactly the same solution as in the continuous time, except that the *equilibrium* is obtained from

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$$f(x_e, u_e) = x_e$$
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