

# Control System

## Lab 9

Palak Singh  
B20EE086

24 October 2022

1. **Objective:** The objective of this laboratory is to design and implement a state-feedback control system that will balance the pendulum in the upright, vertical position.
2. **Apparatus/software required:** MATLAB, Simulink, and Inverted Pendulum experiment setup.
3. **Modeling:**

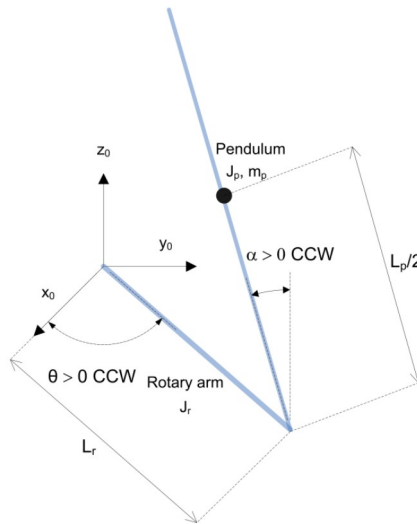


Figure 1: Inverted pendulum conventions

### Non-linear equations of motion:

The Euler-Lagrange method is used to determine the equations of motion. The calculations and obtained results are given by:

$$Q_1 = \tau - B_r \dot{\theta}$$
$$Q_2 = -B_p \dot{\alpha}$$

Substituting the values of  $L$  and  $Q$  and rearranging the equation we get non-linear equation of motion.

$$\begin{aligned}
& \left( m_p L_r^2 + \frac{1}{4} m_p L_p^2 - \frac{1}{4} m_p L_p^2 \cos(\alpha)^2 + J_r \right) \ddot{\theta} - \left( \frac{1}{2} m_p L_p L_r \cos(\alpha) \right) \ddot{\alpha} \\
& + \left( \frac{1}{2} m_p L_p^2 \sin(\alpha) \cos(\alpha) \right) \dot{\theta} \dot{\alpha} + \left( \frac{1}{2} m_p L_p L_r \sin(\alpha) \right) \dot{\alpha}^2 = \tau - B_r \dot{\theta} \\
& - \frac{1}{2} m_p L_p L_r \cos(\alpha) \ddot{\theta} + \left( J_p + \frac{1}{4} m_p L_p^2 \right) \ddot{\alpha} - \frac{1}{4} m_p L_p^2 \cos(\alpha) \sin(\alpha) \dot{\theta}^2 \\
& - \frac{1}{2} m_p L_p g \sin(\alpha) = -B_p \dot{\alpha}.
\end{aligned}$$

#### 4. Linear State Space Model:

$$f_{lin} = f(z_0) + \left( \frac{\partial f(z)}{\partial z_1} \right) \Big|_{z=z_0} (z_1 - a) + \left( \frac{\partial f(z)}{\partial z_2} \right) \Big|_{z=z_0} (z_2 - b)$$

After using the above linearization equations, we obtain the linear state-space model of the form:

$$\begin{aligned}
\dot{x} &= Ax + Bu \\
y &= Cx + Du
\end{aligned}$$

$$x = \begin{bmatrix} 0 \\ \alpha \\ \dot{\theta} \\ \dot{\alpha} \end{bmatrix}$$

Replacing given values of pendulum mass and dimensions the above matrices are given by:

$$A = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & 80.3 & -45.8 & -0.93 \\ 0 & 122 & -44.1 & -1.4 \end{bmatrix}, B = \begin{bmatrix} 0 \\ 0 \\ 83.4 \\ 80.3 \end{bmatrix}$$

$$C = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \end{bmatrix}, D = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

## 5. Balance Control:

### 1. Pole placement using companion matrix method:

18-10-22

CONTROL LAB

INVERTED PENDULUM

$\xi = 0.7$  - Damping Ratio  
 $\omega_n = 4 \text{ rad/sec}$   
 $\alpha < 15^\circ$   
 $V_m < 10V$

$\dot{x} = Ax + Bu$   
 $y = Cx + Du$   
 $x = [\theta \ \dot{\theta} \ \ddot{\theta} \ \ddot{\ddot{\theta}}]^T$

$A = \begin{bmatrix} 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & 80.3 & -45.8 & -0.93 \\ 0 & 122 & -44.1 & -1.4 \end{bmatrix}$

$B = \begin{bmatrix} 0 \\ 0 \\ 83.4 \\ 80.3 \end{bmatrix}$

$p_1 = -\sigma + j\omega_d$   
 $p_2 = -\sigma - j\omega_d$   
 $p_3 = -30$   
 $p_4 = -40$

$\omega_d = \omega_n \sqrt{1 - \xi^2}$   
 $DP = [p_1 \ p_2 \ p_3 \ p_4]^T$

$\omega_d = \omega_n \sqrt{1 - (\xi)^2} = \omega_n \sqrt{1 - (0.7)^2}$   
 $\omega_d = 4 \sqrt{1 - 0.49} = \underline{\underline{2.85}}$   
 $\sigma = \omega_n \xi = 4 \times (0.7) = 2.8$   
 $p_1 = -2.8 + 2.85j$      $p_2 = -2.8 - 2.85j$   
 $DP = \begin{bmatrix} -2.8 + 2.85j & -2.8 - 2.85j & -30 & -40 \end{bmatrix}$   
 Dominant Poles

```

A = [0 0 1 0;
      0 0 0 1;
      0 80.3 -45.8 -0.93;
      0 122 -44.1 -1.4]
B = [0; 0; 83.4; 80.3]

a = poly(A);
Ac = [0 1 0 0;
      0 0 1 0;
      0 0 0 1;
      -a(5) -a(4) -a(3) -a(2) ];

Bc = [0; 0; 0; 1];

T = ctrb(A,B);
Tc = ctrb(Ac, Bc);

W = T*inv(Tc);
Kc = acker(Ac, Bc, dp);
K1 = Kc*inv(W);

```

## 2. From scratch to find the gain matrix:

```

dp = [-2.8+2.85i -2.8-2.85i -30 -40];

Qc = [B A*B (A^2)*B (A^3)*B];
det(Qc)

P = [0 0 0 1]*inv(Qc);

Pc = [P; P*A; P*(A)^2; P*(A)^3];
b = poly(dp);

a = a(2:5);
b = b(2:5);

K2 = flip(b-a)*Pc

```

It is observed that,  $K = K1 = K2 = [5.1399 \ 28.0233 \ 2.7117 \ 3.1701]$

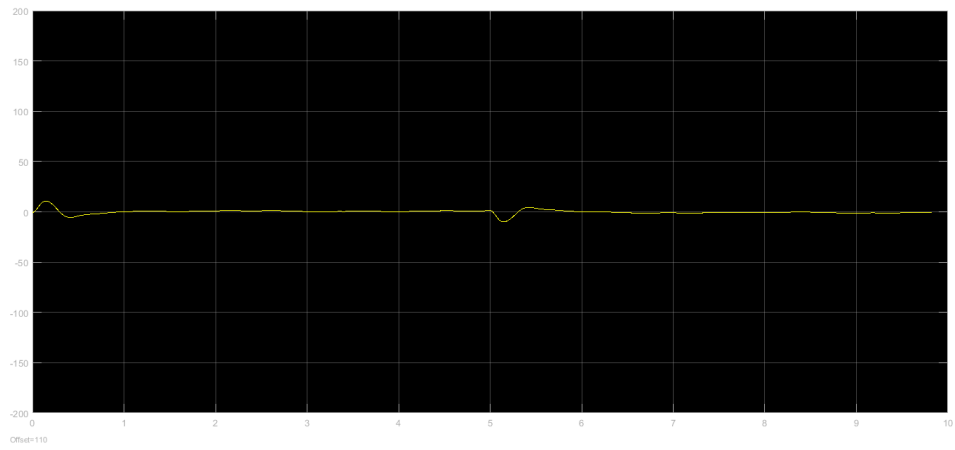


Figure 2:  $\alpha$

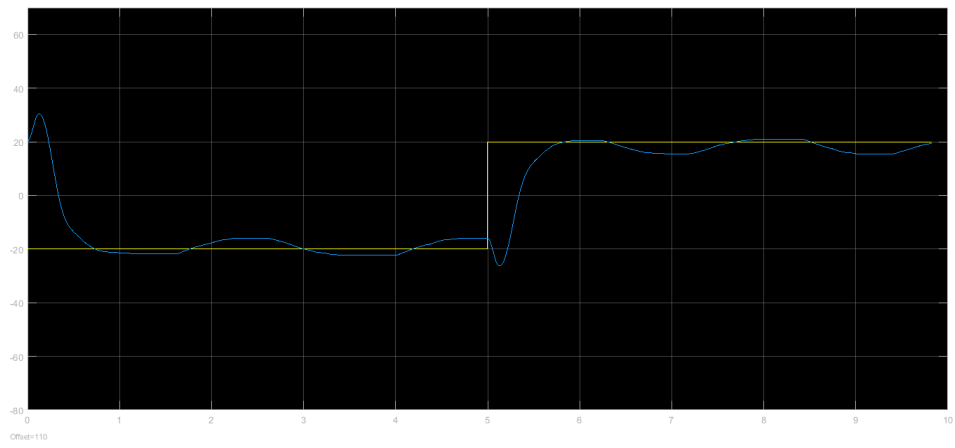


Figure 3:  $\theta$

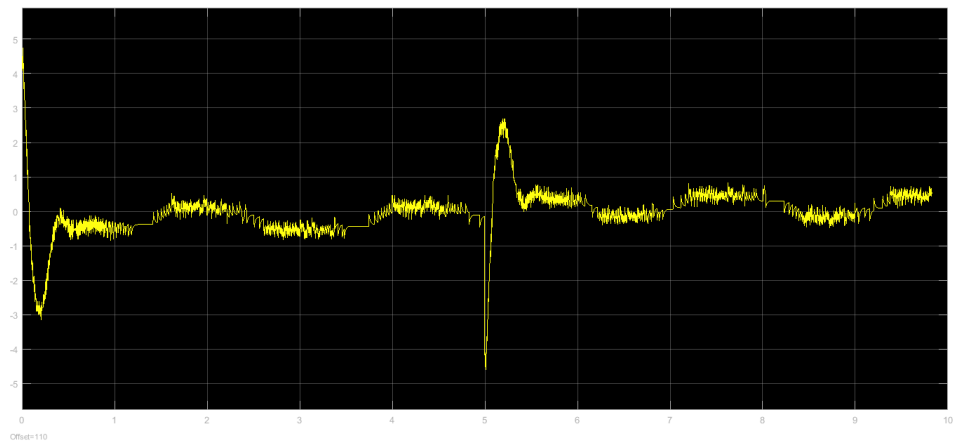


Figure 4: Motor voltage

## 6. Swing up control:

~~K<sub>p</sub> = 1.5~~ Swing up control non linear control

~~K<sub>p</sub>~~  $\dot{\alpha} =$

$$E_p = \frac{1}{2} m_p g L_p (1 - \cos \alpha)$$

$$E_k = \frac{1}{2} J_p \dot{\alpha}^2$$

$$\text{Total Energy} = E_p + E_k = \frac{1}{2} m_p g L_p (1 - \cos \alpha) + \frac{1}{2} J_p \dot{\alpha}^2$$

$$\tau = \frac{\eta_g K_g \eta_m K_t (V_m - K_g K_m \dot{\theta})}{R_m}$$

$V_m - K_g K_m \dot{\theta} = 5$   
 $\eta_g = 0.69$   
 $K_g = 70$   
 $\eta_m = 0.9$   
 $K_t = 0.00768$   
 $R_m = 2.6$

$$\tau = \frac{0.69 \times 70 \times 0.9 \times 0.00768 \times 5}{2.6}$$

$$\tau = 0.642$$

~~Handwritten scribbles~~

$$\mu_{\max} = \frac{\tau}{m_r l_r} = \frac{0.642}{(0.257)(0.216)}$$

$m_r = 0.25$   
 $l_r = 0.216$

$$\mu_{\max} = 11.565 = 11.565 \text{ m/s}^2$$

Upward Energy  $\rightarrow \alpha = 0$

$$\text{Upward Energy} = \frac{1}{2} J_p \dot{\alpha}^2 = 0$$

Downwards Energy  $\rightarrow \alpha = 180^\circ$

$$= \underbrace{\frac{1}{2} J_p \dot{\alpha}^2}_0 + \frac{1}{2} \times (0.127)(9.8)(0.337)(1 - (-1))$$

$$= \underline{\underline{0.4894}}$$

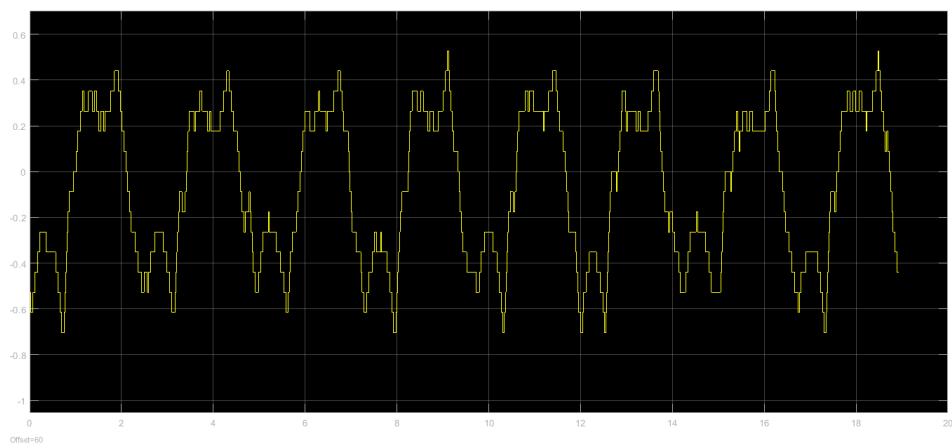


Figure 5:  $\alpha$

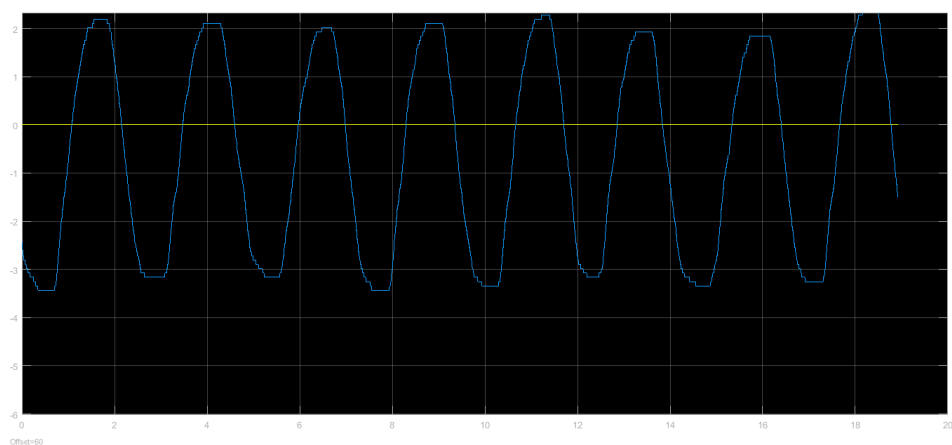


Figure 6:  $\theta$



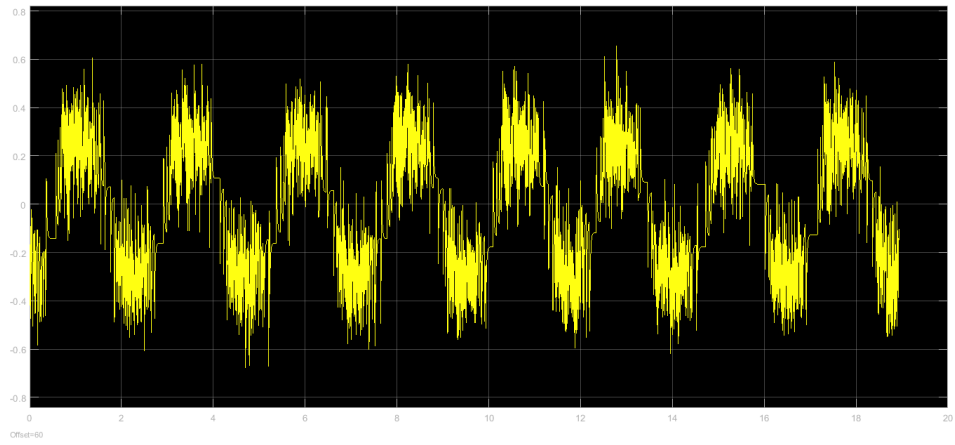


Figure 7: Motor voltage