

# Control System Lab

## Experiment 8: Halfquadrotor

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October 17, 2022

### 1 Objective

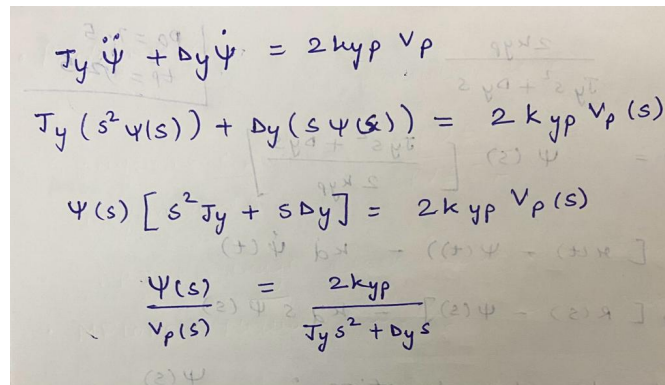
To estimate viscous damping coefficients, the cross thrust gain parameters and to design and simulate PD control system.

### 2 Theory

The Quanser Aero Experiment experiment can be configured as a half-quadrotor system. By changing the direction and speed of the rotors, users can change the yaw axis angle. Unmanned quadrotor vehicles are used for wide-variety of applications. Using a tethered half-quadrotor systems allows students and researchers to focus on the modeling, control, and parameter estimation in yaw-axis motion of quadrotors, which can then be applied to full quadrotor system.

### 3 Modeling

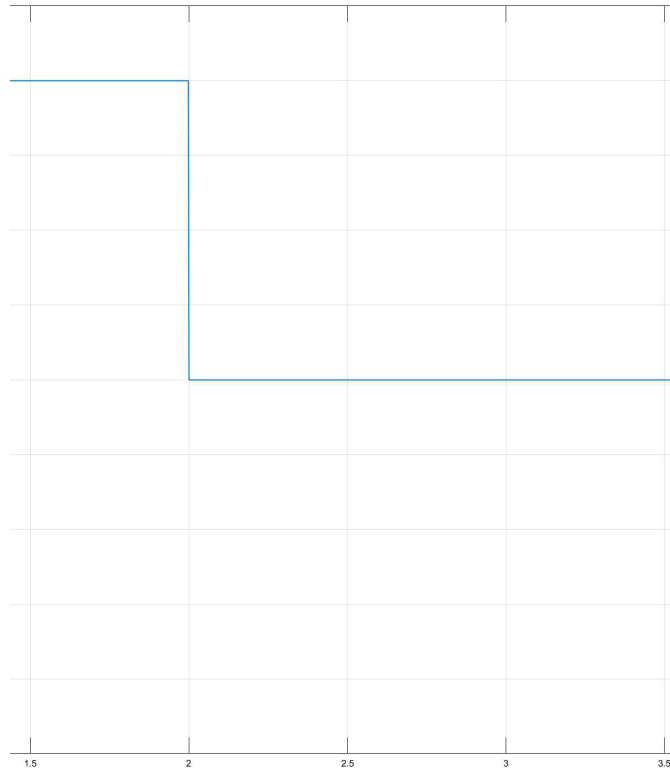
#### 3.1 Transfer function model

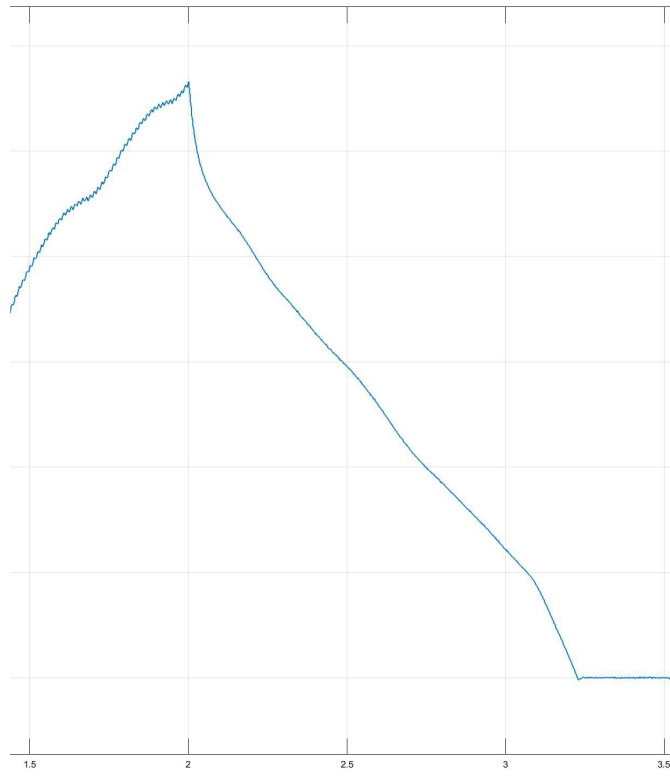


Handwritten derivation of the transfer function for a half-quadrotor system:

$$J_y \ddot{\psi} + D_y \dot{\psi} = 2k_{yp} v_p$$
$$J_y (s^2 \psi(s)) + D_y (s \psi(s)) = 2k_{yp} v_p(s)$$
$$\psi(s) [s^2 J_y + s D_y] = 2k_{yp} v_p(s)$$
$$\frac{\psi(s)}{v_p(s)} = \frac{2k_{yp}}{J_y s^2 + D_y s}$$

### 3.2 Estimating viscous damping coefficients



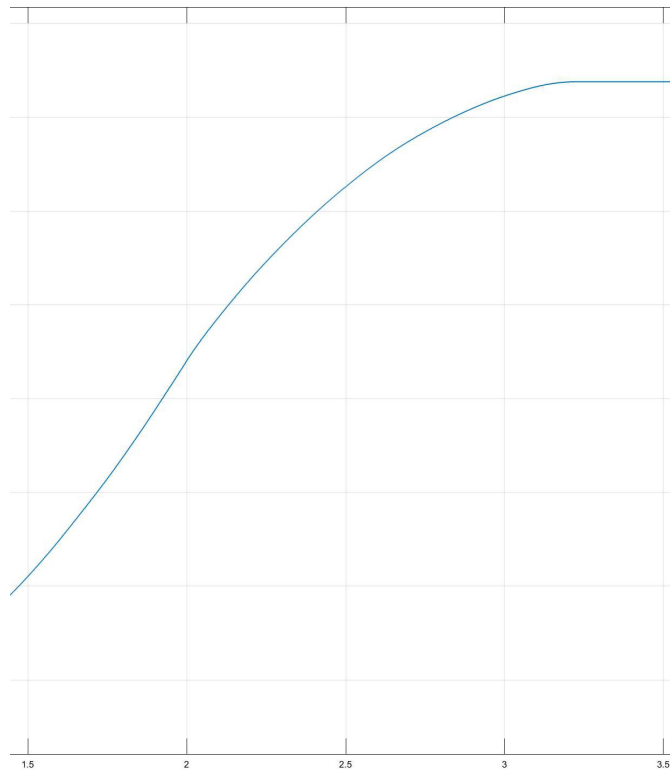


From the above plot we note the Speed and time to calculate the time constant.

$$\begin{aligned}
 \omega_0 &= A_y = 2.789 \\
 \omega_1 &= \frac{37}{100} \times \omega_0 = 1.03193 \\
 t_1 &= 2.729 \\
 t_0 &= 2 \\
 \text{So, } \tau &= 2.729 - 2 = 0.729 \text{ s} \\
 \text{Now, Damping is } \rho_y &= \frac{Tt}{\tau} \\
 \rho_y &= \frac{0.022}{0.729} \Rightarrow \rho_y = 0.0301
 \end{aligned}$$

### 3.3 Estimating the Cross-thrust gain parameters

Plots of yaw step response from step voltage.



the cross thrust gain of the yaw due to the pitch rotor is .

$$k_{yp} = \frac{T_y \frac{\Delta w_y}{\Delta t} + \Delta y \Delta w_p}{V_p}$$

$$V_p = 20 \text{ V}$$

$$\Delta w_y = -4$$

$$\Delta t_y = 12$$

$$k_{yp} = \frac{0.0220 \times \frac{-4}{12} + 0.0301 \times -4}{20}$$

$$= \frac{-0.00733 - 0.1204}{20} = -0.0063865$$

## 4 PD Control

### 4.1 Background

New,  $\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}u$   
 $\dot{\mathbf{y}} = \mathbf{C}\mathbf{x} + \mathbf{D}u$

(2)  $\mathbf{x}_1 = \psi$        $\mathbf{x}_2 = \dot{\psi}$   
 $\dot{\mathbf{x}}_1 = \dot{\psi}$        $\dot{\mathbf{x}}_2 = \ddot{\psi}$   
 $\dot{\mathbf{x}}_1 = \mathbf{x}_2$   
 $\dot{\mathbf{x}}_2 = -\frac{D_y}{J_y} \dot{\psi} + \frac{2k_{yp}}{J_y} V_p$   
 $\dot{\mathbf{x}}_2 = -\frac{D_y}{J_y} \mathbf{x}_2 + \frac{2k_{yp}}{J_y} V_p$

can be written as:

$$\begin{bmatrix} \dot{\mathbf{x}}_1 \\ \dot{\mathbf{x}}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ 0 & -\frac{D_y}{J_y} \end{bmatrix} \begin{bmatrix} \psi \\ \dot{\psi} \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{2k_{yp}}{J_y} \end{bmatrix} V_p$$

$\mathbf{A} = \begin{bmatrix} 0 & 1 \\ 0 & -\frac{D_y}{J_y} \end{bmatrix}$        $\mathbf{B} = \begin{bmatrix} 0 \\ \frac{2k_{yp}}{J_y} \end{bmatrix}$   
 $\mathbf{y} = \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} \psi \\ \dot{\psi} \end{bmatrix}$        $\mathbf{C} = \begin{bmatrix} 1 & 0 \end{bmatrix}$   
 $\mathbf{D} = \begin{bmatrix} 0 \end{bmatrix}$

$$\frac{\Psi(s)}{U(s)} = \frac{2k_{yp}}{J_y s^2 + D_y s} \quad \left. \begin{array}{l} p_0 = 7.5 \\ t_p = 1.25 \end{array} \right\}$$

$$\Rightarrow U(s) = \Psi(s) \left[ \frac{J_y s^2 + D_y s}{2k_{yp}} \right]$$

$$u = k_p [x(t) - \psi(t)] - k_d \dot{\psi}(t)$$

$$U(s) = k_p [R(s) - \Psi(s)] - k_d s \Psi(s)$$

closed loop transfer function:  $\frac{\Psi(s)}{R(s)}$

$$\Psi(s) \left[ \frac{J_y s^2 + D_y s}{2k_{yp}} \right] = \Psi(s) [-k_p - s k_d] + k_p R(s)$$

$$\frac{\Psi(s)}{R(s)} = \frac{k_p}{k_p + k_d s + \frac{D_y s}{2k_{yp}} + \frac{J_y s^2}{2k_{yp}}}$$

$$\frac{\Psi(s)}{R(s)} = \frac{2k_{yp} k_p}{J_y s^2 + s(D_y + 2k_{yp} k_d) + 2k_{yp} k_p}$$

$$= \frac{\frac{2k_{yp} k_p}{J_y} \times \frac{D_y}{D_y}}{s^2 + s \left( \frac{D_y}{J_y} + \frac{2k_{yp} k_d}{J_y} \right) + \frac{2k_{yp} k_p}{J_y}}$$

Comparing with:

$$s^2 + 2\zeta \omega_n s + \omega_n^2 = 0$$

standard second order transfer function:  $f^n = \frac{\omega_n^2}{s^2 + 2\zeta \omega_n s + \omega_n^2}$

$$2\zeta\omega_n = \frac{D_y}{J_y} + \frac{2k_{yp}k_d}{J_y} \quad (1)$$

$$\omega_n^2 = \frac{2k_{yp}k_p}{J_y} \quad (2)$$

$$\text{Also, } \omega_n = \frac{\pi}{t_p \sqrt{1-\zeta^2}}$$

$$\zeta = \frac{-\ln\left(\frac{p_0}{100}\right)}{\sqrt{\left[\ln\left(\frac{p_0}{100}\right)\right]^2 + \pi^2}}$$

$$\text{Given, } t_p = 1.25 \\ p_0 = 7.5\%$$

$$\text{So, } \zeta = \frac{-\ln\left(\frac{0.075}{100}\right)}{\sqrt{\left[\ln\left(\frac{0.075}{100}\right)\right]^2 + \pi^2}} = \frac{+7.195}{\sqrt{51.7143 + 9.869}}$$

$$\zeta = 0.636$$

$$\omega_n = \frac{\pi}{1.25 \sqrt{1-(0.636)^2}} = 3.26 \text{ rad/s}$$

Using (2)

$$(3.26)^2 = \frac{2 \times k_{yp} \times k_p}{J_y}$$

$$\Rightarrow J_y K_p = 5.3138 \frac{T_y}{k_{yp}}$$

Using (1)

$$2(0.636)(3.26) = \frac{D_y}{J_y} + \frac{2k_{yp}k_d}{J_y}$$

$$k_p = \frac{c \omega_n^2}{k} = \frac{J_y \times \omega_n^2 \times \Delta y}{\Delta y \times 2k_{yp}}$$

$$= \frac{J_y \Delta y \omega_n^2}{2k_{yp}}$$

$$k_d = \frac{2 \times J_y \zeta \omega_n - 1}{\Delta y}$$

$$\Rightarrow k_p = \frac{0.022 \times (3.26)^2}{2 \times (-0.0063865)} = -18.3$$

$$\text{and } k_d = \frac{2 \times 0.022 \times 0.63 \times 3.26 - 1}{0.0301} = \frac{4.785}{0.0301}$$

$$\Rightarrow 4.14672 J_y = \Delta y + 2k_{yp}k_d$$

$$\Rightarrow k_d = \frac{4.14672 J_y - \Delta y}{2k_{yp}}$$

## 4.2 PD Control design and Simulation

Desired closed loop specification:

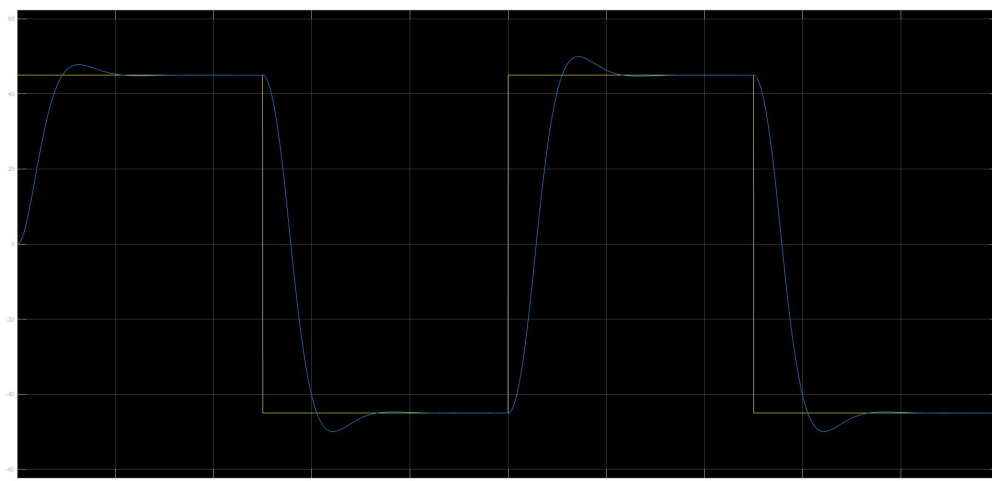
1. Steady state error  $\leq 2\%$
2. Peak time  $\leq 2s$
3. Percent overshoot  $\leq 7.5\%$
4. No actuator saturation,  $|V_y| \leq 24V$  and  $|V_p| \leq 24V$

This model uses the QUARC PID block to implement the PV control for the pitch and yaw axes. The Quanser AERO Half-Quadrotor Model subsystem implements the transfer function described earlier. We will design the PV gains for the peak time  $t_p = 1.25$  rad/s and the overshoot of  $PO = 7.5\%$ . We use a lower peak time specification to generate the PV gain to ensure the peak time criteria above is satisfied.

### Response analysis

Yaw angle





Pitch-yaw motor voltage

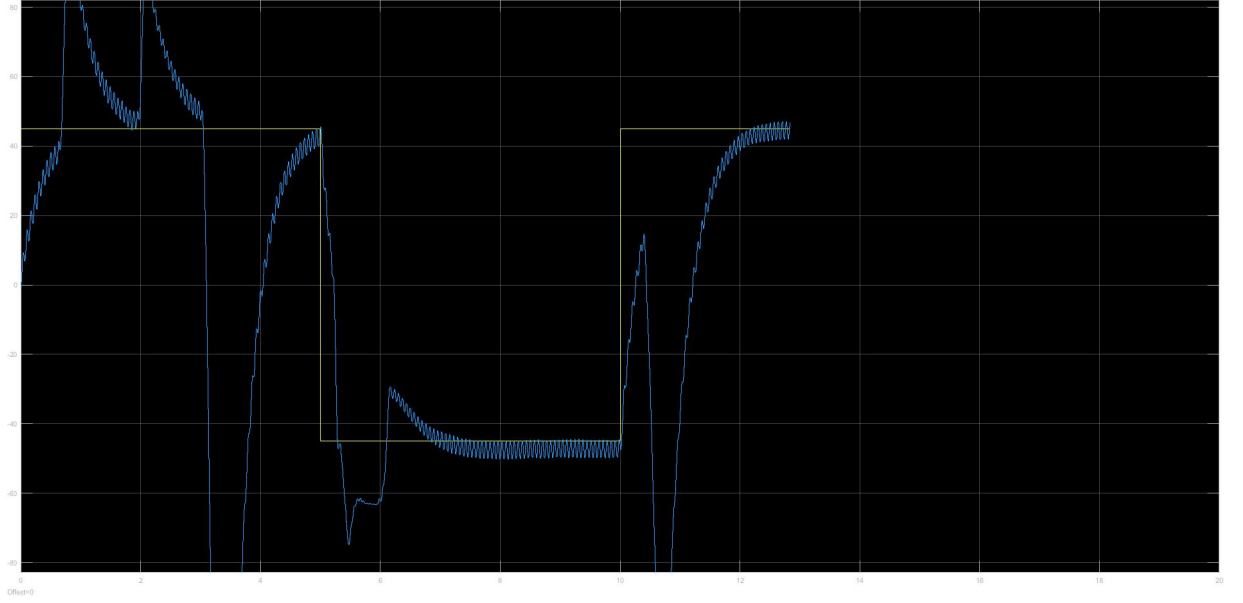


The peak time, overshoot, and steady-state error of the simulated response are:

1. Steady state error =  $0 \leq 2^\circ$
2. Peak time =  $11.4 - 10 = 1.4\text{s} \leq 2\text{s}$
3. Percent overshoot =  $(49.9 - 45)/90 = 5.4\text{percent} \leq 7.5\text{percent}$
4. No actuator saturation ,  $|V_y| \leq 24\text{V}$  and  $|V_p| \leq 24\text{V}$

### 4.3 Running PD on system

yaw position(degree)



Here, we notice that as we try to move the motor manually it again try to rotate in the opposite direction to chase its path. The yellow line represents its desired path whereas the blue line represents its actual path.

## 5 Results and Discussions

The observations are probably due to the unmodeled friction (e.g. Coulomb friction) about the yaw axis. Increasing the proportional gain, introducing an integrator, or using a more advanced friction-compensation scheme could minimize this. Lastly, the motors do get saturated but only for a short instant so this is still acceptable. The saturation blocks prevent higher voltages as well.