

## 2 L2: Relative Plane Motion Analysis - Velocity

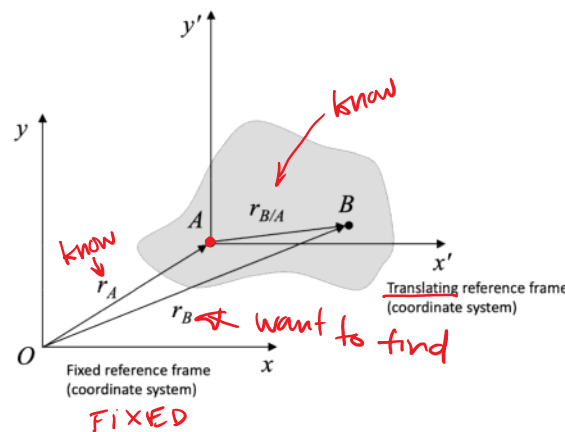
### Readings

### 2.1 Objective

To describe the motion of a point on a rigid body using a translating reference frame attached to the body.

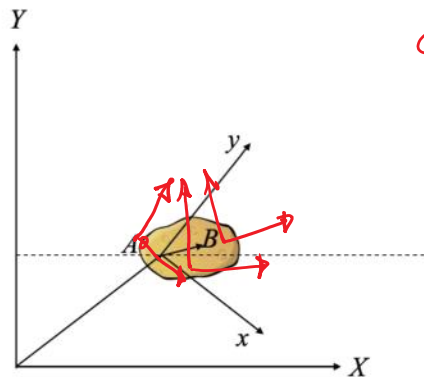
### 2.2 Fixed and Translating Reference Frames

In this section we consider a frame  $x'y'$  attached to a body at a point  $A$  as if by a pin – i.e.: translating but NOT rotating with the body.  $xy$  is a fixed frame (attached to ground).



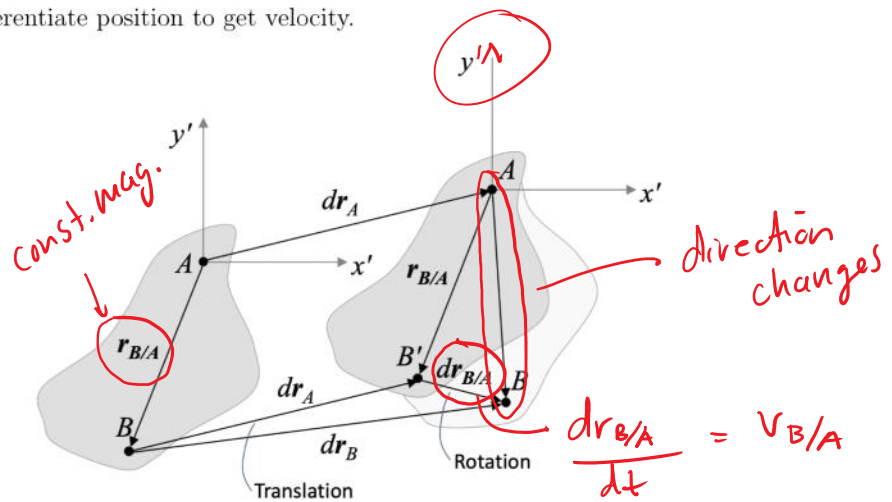
|  |   |  |   |   |
|--|---|--|---|---|
| $r_A$  | + | $r_{B/A}$  | = | $r_B$   |
| Describes the location of the <u>translating reference frame</u> $x'y'$ with respect to the fixed $xy$ frame | + | Describes the location of point <u>B</u> with respect to <u>translating reference frame</u> $x'y'$ | = | Describes the location of point B with respect to the <u>fixed frame</u> $xy$ |

IMPORTANT NOTE: "with respect to" means "measured from". BUT we must make sure we are using the **same axis directions** to do the measurements.



choose a point  
for translating  
frame that  
has known  
motion

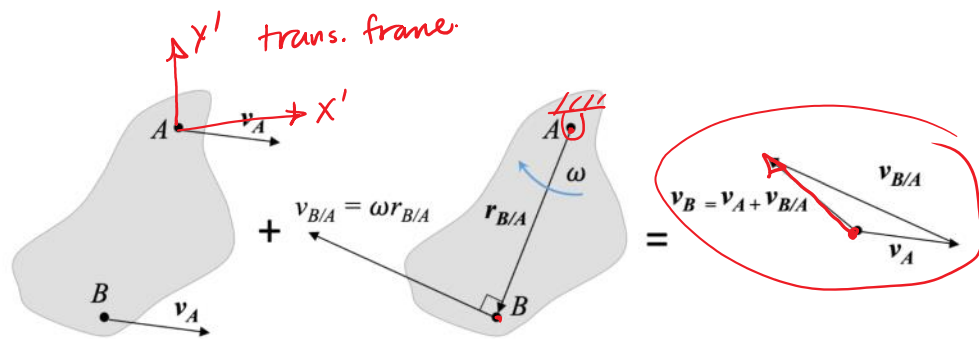
As before, we differentiate position to get velocity.



Since the body is rigid (i.e. does not change size), the magnitude of  $\mathbf{r}_{B/A}$  must be constant. Therefore,  $\mathbf{v}_{B/A} = \frac{d\mathbf{r}_{B/A}}{dt}$  is due only to the **rotation** of point  $B$  about point  $A$ . That is, the motion follows Chasles' theorem:

change in dir.

$$\mathbf{v}_B = \mathbf{v}_A + \boldsymbol{\omega} \times \mathbf{r}_{B/A}$$

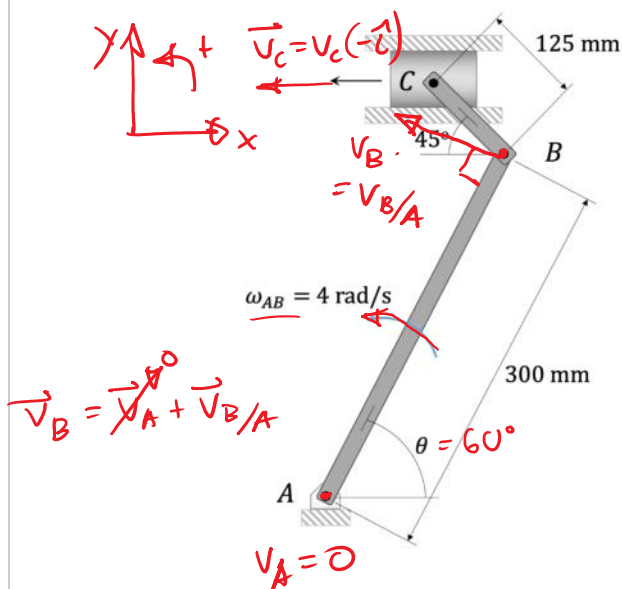


Translation

Rotation about  
the base point A
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### 2.2.1 Example

Find  $\omega_{BC}$  and  $v_C$  at the instant shown.  $\theta = 60^\circ$ .



1. choose coord.
2. draw vector diagram

\* KINEMATIC  
CONSTRAINTS\*

3. write eqns
4. solve

use  
your  
brain

Given:

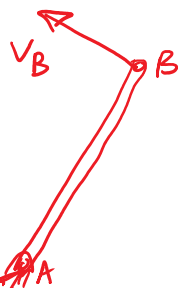
$$r_{AB} = 0.3 \text{ m}$$

$$r_{BC} = 0.125 \text{ m}$$

$$\theta = 60^\circ$$

$$\omega_{AB} = 4 \text{ rad/s}$$

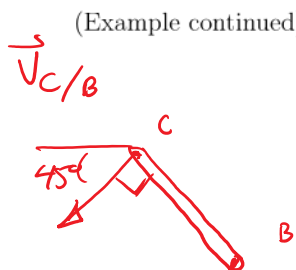
$$v_A = 0$$



$$\vec{v}_B = \vec{\omega}_{AB} \times \vec{r}_{B/A}$$

$$\begin{aligned} \|\vec{v}_B\| &= \|\vec{\omega}_{AB}\| \|\vec{r}_{B/A}\| \\ &= |4| |0.3| \text{ m/s} \\ &= 1.2 \text{ m/s} \end{aligned}$$

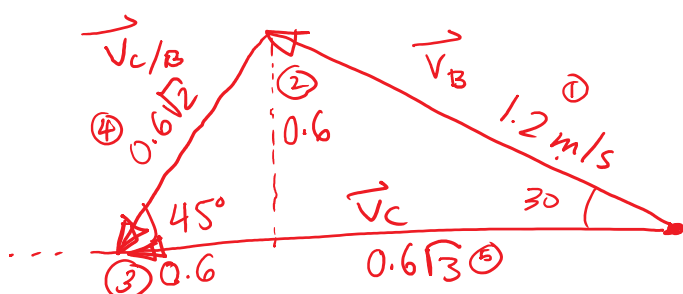
(Example continued)



$$\vec{v}_{C/B} = \vec{\omega}_{BC} \times \vec{r}_{C/B}$$

$$\vec{v}_C = \vec{v}_B + \vec{v}_{C/B} = \vec{v}_B + \vec{\omega}_{BC} \times \vec{r}_{C/B} = v_C (-\hat{i})$$

BY VECTOR DIAGRAM:



$$\vec{v}_C = 0.6 + 0.6\sqrt{3} = 1.64(-\hat{i}) \text{ m/s}$$

$$\cos 30 = \frac{\sqrt{3}}{2}$$

$$\sin 30 = \frac{1}{2}$$

$$|\vec{v}_{C/B}| = 0.6\sqrt{2}$$

$$= |\vec{\omega}_{BC}| |\vec{r}_{C/B}|$$

$$|\vec{\omega}_{BC}| = \frac{0.6\sqrt{2} \text{ m/s}}{0.125 \text{ m}} = 6.7 \text{ rad/s} \quad \curvearrowright$$

BY MATH/SYS.  
OF EQNS

(\vec{v}\_A = 0, \text{ pinned})

$$\vec{v}_C = v_C \hat{i} = \vec{v}_B + \vec{v}_{C/B} = \vec{\omega}_{AB} \times \vec{r}_{B/A} + \vec{\omega}_{BC} \times \vec{r}_{C/B}$$

$$v_C \hat{i} = \underbrace{4 \text{ rad/s}}_{\vec{\omega}_{AB}} \hat{k} \times \underbrace{0.3 \text{ m} \left( \frac{1}{2} \hat{i} + \frac{\sqrt{3}}{2} \hat{j} \right)}_{\vec{r}_{B/A}} + \omega_{BC} \hat{k} \times 0.125 \text{ m} \left( -\frac{1}{\sqrt{2}} \hat{i} + \frac{1}{\sqrt{2}} \hat{j} \right)$$

$$v_C \hat{i} = \left[ -0.6\sqrt{3} \hat{i} + 0.6 \hat{j} - \omega_{BC} \left( \frac{0.125}{\sqrt{2}} \right) \hat{i} - \omega_{BC} \left( \frac{0.125}{\sqrt{2}} \right) \hat{j} \right]$$

By components:

$$\hat{i}: v_C = -0.6\sqrt{3} - \omega_{BC} \left( \frac{0.125}{\sqrt{2}} \right) \text{ m/s} \quad \textcircled{A}$$

$$\hat{j}: 0 = 0.6 - \omega_{BC} \left( \frac{0.125}{\sqrt{2}} \right) \text{ m/s} \quad \textcircled{B}$$

$$\textcircled{B} \quad \omega_{BC} = \frac{0.6\sqrt{2}}{0.125} = 6.7 \text{ rad/s} \quad \vec{\omega}_{BC} = 6.7 \text{ rad/s } \hat{k}$$

$$(\vec{\omega}_{BC} = \omega_{BC} \hat{k})$$

$$\textcircled{A} : \quad v_c = -0.6\sqrt{3} - 0.6 \text{ m/s} = -1.64 \text{ m/s}$$

$$\vec{v}_c = -1.64 \text{ m/s } \hat{u}$$