

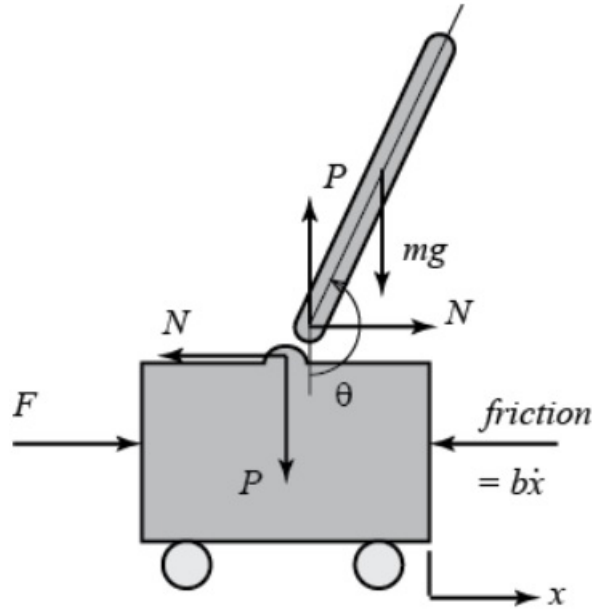
# Inverted Pendulum

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## 1 Mathematical Modelling

Let the mass of the cart be  $M$  and that of the rod be  $m$ . The friction coefficient of the road (influencing the cart wheels) is  $b$  and the external force on the cart is  $F$ .

The free-body diagram of the cart can be drawn as follows:



The EOM (Equations of motion) are:

$$M\ddot{x} + b\dot{x} + N = F \quad (1)$$

$$N = m\ddot{x} + m\ddot{\theta}\cos\theta - m\dot{\theta}^2\sin\theta \quad (2)$$

$$P\sin\theta + N\cos\theta - mg\sin\theta = m\ddot{\theta} + m\ddot{x}\cos\theta \quad (3)$$

$$-Pl\sin\theta - Nl\cos\theta = I\ddot{\theta} \quad (4)$$

Using equations 1 and 2 we get:

$$(M + m)\ddot{x} + b\dot{x} + ml\ddot{\theta}\cos\theta - ml\dot{\theta}^2\sin\theta = F \quad (5)$$

Using equations 3 and 4 we get:

$$(I + ml^2)\ddot{\theta} + mgl\sin\theta = -ml\ddot{x}\cos\theta \quad (6)$$

The dynamics of this system are non-linear and we need to linearize the system in-order for the subsequent control design techniques to be applicable. More about linearization of dynamical systems can be found in this lecture slide: *System Analysis and Control*.

The equilibrium point we choose to linearize the system about, is where  $\theta = \pi$ .

Let the angle of disturbance be  $\phi$ . Thus  $\theta = \pi + \phi$ .

It follows that  $\cos\theta = -1$  and  $\sin\theta = -\phi$ .

Also  $\dot{\theta}^2 = \dot{\phi}^2 \approx 0$

Note: The angle of disturbance is with regard to the equilibrium position ( $\theta = \pi$ ).

With these substitutions, we get the following equations:

$$(I + ml^2)\ddot{\phi} - mgl\phi = ml\ddot{x} \quad (7)$$

$$(M + m)\ddot{x} + b\dot{x} - ml\ddot{\phi} = F \quad (8)$$

This completes the mathematical analysis of the system.