

Tutorial # 4.

①

Overview: Disturbance observer for PI controller.

In the design, we choose two desired closed-loop poles to calculate the controller and estimator gain.

- α_1 is the closed-loop pole for proportional control.

$$K_1 = \frac{\alpha_1 - a}{b}$$

- α_2 is the closed-loop pole for the estimator:

$$K_2 = \frac{\alpha_2}{b}$$

model:

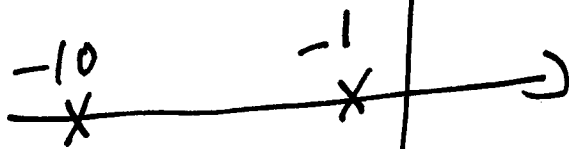
$$G(s) = \frac{b}{s+a}.$$

(2)

$$Q1: \quad G(s) = \frac{e^{-0.001s}}{(s+1)(s+10)}.$$

In order to design the PI controller, for disturbance observer, we simplify the model to get a first order model.

Two poles:



We will neglect the pole

at -10. We write

$$G(s) = \frac{e^{-0.001s}}{(s+1) \times 10 \left(\frac{1}{10}s + 1\right)}$$

We approximate

$$\frac{1}{\frac{1}{10}s + 1} \approx 1 \text{ and } e^{-0.001s} \approx 1$$

The first order model becomes

$$G(s) \approx \frac{0.1}{s+1}.$$

(3)

$$a = 1, \quad b = 0.1.$$

$$K_1 = \frac{\alpha_1 - a}{b} = \frac{2a - a}{b} = \frac{a}{b} = 10.$$

$$K_2 = \frac{\alpha_2}{b} = \frac{3a}{b} = 30$$

We evaluate the closed-loop performance through simulation. Note that in the simulation, we need to use the original system.

We simulate the closed-loop response without constraints first to find the

$$u^{\max} \Rightarrow 40$$

$$u^{\min} \Rightarrow -12.$$

Then we simulate again with the

$$\text{constraints: } u^{\max} = 40 \times 0.85$$

$$u^{\min} = -12 \times 0.85.$$

$$Q_2: G(s) = \frac{2e^{-0.5s}}{(s+0.1)(s+10)^2}$$

(4)

The system has three poles and a delay. The dominant pole is -0.1 . The time delay is small, relative to the dominant time constant which is $10 \approx \frac{1}{0.1}$. We neglect the two poles at -10 and the time delay.

$$\frac{2e^{-0.5s}}{(s+0.1)(s+10)^2} \approx \frac{2e^{-0.5s}}{100(s+0.1)\left(\frac{1}{10}s+1\right)^2}$$

$$\approx \frac{0.02}{s+0.1}$$

$$\frac{e^{-0.5s}}{1} \approx 1$$

$$\frac{1}{\left(\frac{1}{10}s+1\right)^2} \approx 1$$

$$K_1 = \frac{a_1 - a}{b}$$

$$= \frac{2a - a}{b} = \frac{a}{b} = \frac{0.1}{0.02} = 5$$

$$K_2 = \frac{3a}{b} = \frac{0.1 \times 3}{0.02} = 15$$

without constraints,

(5)

$$u^{\max} \Rightarrow 22$$

$$u^{\min} \Rightarrow -16$$

For disturbance rejection, the u^{\min} is ~~almost~~ required to achieve zero steady-state ^{error} compensation. We can experiment and find out if we constrain the $u^{\min} = -16 * 0.85$, then the disturbance rejection is not good. However, the system has no oscillation.

⑥

Overview PID controller with disturbance observer. / second order model

$$G(s) = \frac{b}{s^2 + a_1 s + a_0}$$

Design for PD controller. The closed-loop performance is specified with the polynomial: $s^2 + 2\zeta\omega_n s + \omega_n^2$

$\zeta = 0.707$. ω_n is selected by the user.

$$K_1 = \frac{\omega_n^2 - a_0}{b}, \quad K_2 = \frac{2\zeta\omega_n - a_1}{b}$$

the derivative filter is

$$\tau_f = \beta \tau_D = 0.1 \frac{K_2}{K_1}$$

$\beta = 0.1$ for implementation.

The estimated gain K_3 is the same as before, (7)

$$K_3 = \frac{\alpha_3}{b}$$

where $-\alpha_3$ is the desired closed-loop pole.

Implementation:

$$\hat{d}(t) = \hat{z}(t) - K_3 (\dot{y}_f(t))$$

$$u(t) = -K_1 (y(t) - r(t)) - K_2 \dot{y}_f(t) - \hat{d}(t)$$

saturate $u(t)$ if it exceeds limit.

$$\begin{aligned} \dot{\hat{z}}(t) = & -\alpha_3 \hat{z}(t) + K_3 (a_1 - \alpha_3) \dot{y}(t) \\ & + K_3 a_0 (y(t) - r(t)) - \alpha_3 u(t) \end{aligned}$$

(8)

$$Q3: G(s) = \frac{2e^{-0.01s}}{(s-1)(s+1)} \approx \frac{2}{s^2-1}$$

$$a_1 = 0, a_0 = -1, b = 2. \quad e^{-0.01s} \approx 1$$

$$\zeta = 0.707, \omega_n = 3.$$

$$K_1 = \frac{\omega_n^2 - a_0}{b} = \frac{9 + 1}{2} = 5$$

$$K_2 = \frac{2 \times 0.707 \times 3 - 0}{b} = 4.242.$$

$$K_3 = \frac{\alpha_3}{b} = \frac{4}{2} = 2.$$

$$\tau_D = \frac{K_2}{K_1} = \frac{4.242}{5} = 0.8484.$$

$$\tau_f = \tau_D \times 0.1 = 0.08484.$$