

# INTRO TO FINITE ELEMENT METHODS

A. DONEV, COURANT (1)

We will consider 1D elliptic  
BVPs (Sturm-Liouville problems)

$$-\frac{d}{dx} \left[ a(x) \frac{du}{dx} \right] + b(x)u = f$$
$$x \in [0, 1]$$

$$a(x) > 0, \quad b(x) \geq 0$$

And  $\left. \begin{array}{l} u(0) = \alpha \\ u(1) = \beta \end{array} \right\} \text{essential BCs}$

The key idea of series methods is to approximate  $u$  with a linear combination of basis functions:

(2)

$$\left\{ \begin{array}{l} u_m = \psi_0(x) + \sum_{l=1}^m \gamma_l \psi_l(x) \\ \psi_0 \text{ satisfies inhomogeneous BCs} \\ \psi_l \text{ satisfies homogeneous BCs, } l \geq 1 \end{array} \right.$$

We want  $u_m(x)$  to approximate  $u(x)$

Plug  $u_m$  into PDE to get  
the residual or defect :

③

$$d_m(x) = - \frac{d}{dx} \left( a \frac{du_m}{dx} \right) + b u_m - f$$

We can minimize  $d_m(x)$  in a few ways

① Collocation : Impose  $d_m(x_j) = 0$

at a grid of points  $x_j$ .

E.g. use Chebyshev grid and polynomials

② Minimize some norm of  $d_m(x)$   
over the  $x$ 's

Imagine that

$$d_m(x) \in \overset{\circ}{H}_m = \text{Span} \{ \psi_1, \dots, \psi_m \} \quad (4)$$

↑  
linear subspace

Then  $d_m(x)$  being zero is equivalent to it being orthogonal to all non-zero elements of  $\overset{\circ}{H}_m$ , i.e.

$$\langle d_m, \psi_k \rangle = 0, \quad k=1, \dots, m$$

FOR SOME INNER PRODUCT

This is the Galerkin equations

We focus now on Galerkin methods for second-order BVPs (5)

$$d_m = - \left[ (a\psi_0)' + \sum \gamma_e (a\psi_e)' \right] + b \left[ \psi_0 + \sum \gamma_e \psi_e \right] - f$$

Now compute  $\langle d_m, \psi_k \rangle$  to  
get a linear system of equations  
for the  $\gamma_e$ 's.

$$\sum_l y_l \left[ \overset{\text{Matrix}}{\langle -(a\psi'_l)', \psi_k \rangle} + \langle \underbrace{b\psi_l}_{\text{vector}}, \psi_k \rangle \right] \quad (6)$$

$$= \boxed{\overset{\leftarrow}{A} \overset{\rightarrow}{y} = \overset{\rightarrow}{\delta}} =$$

$$\langle f, \psi_k \rangle = \left[ \langle -(a\psi'_0)', \psi_k \rangle + \langle b\psi_0, \psi_k \rangle \right]$$

From now on assume homogeneous  
BCs so that  $y_0 = 0, \psi_0 = 0$

(7)

$$A_{k,l}^{(a)} = \langle -(a\psi_l')', \psi_k \rangle$$

Let's integrate by parts

$$\langle u, w \rangle = \int_0^1 \underbrace{u(\bar{r})}_{\in L_2} w(\bar{r}) d\bar{r}$$

$$- \int_0^1 (a(\bar{r}) \psi_l'(\bar{r}))' \psi_k(\bar{r}) d\bar{r} =$$

Boundary  
terms  
||  
0

$$+ \int a(\bar{r}) \psi_l'(\bar{r}) \psi_k'(\bar{r}) d\bar{r}$$

Finally we get

$$\overleftrightarrow{A} \vec{p} = \vec{\delta}$$

(8)

$$A_{kl} = \int_0^1 \left[ a(\bar{r}) \psi'_l(\bar{r}) \psi'_k(\bar{r}) + b(\bar{r}) \psi_l(\bar{r}) \psi_k(\bar{r}) \right] d\bar{r}$$

↖  
symmetric matrix

$$\delta_k = \int_0^1 \psi(\bar{r}) \psi_k(\bar{r}) d\bar{r} + \text{boundary condition terms from } \psi_0$$

In finite element circles the matrix  $\overleftrightarrow{A}$  is called the stiffness matrix. It is SPD for  $a > 0$



Note that because we integrated by parts now  $\varphi$ 's need to ⑨ only be piecewise once differentiable and don't have to be twice differentiable. This gives us more flexibility in choosing the basis functions.

↪  
 $A_{..}$  is symmetric positive semidefinite  
(so we can use Cholesky)

Proof: Define

$$y = X^T A X = \sum_{k,l} x_k A_{kl} x_l$$

(10)

$$\underbrace{x_k A_{kl} x_l}_{\text{Implied summation over } k, l} = \int a (x_l \psi_l)' (x_k \psi_k)' d\bar{x} \\ = \int a (\tilde{\varphi}')^2 d\bar{x} \geq 0$$

where  $\tilde{\varphi} = \sum_l x_l \psi_l$

Note : If we had  $\dot{H}^0$  be an infinite dimensional space (say  $L_2$ ) then a solution of the Galerkin equations is a weak solution of the PDE

# Finite Element Basis functions

11

Up to now everything was general and really came from analysis. The basis functions could be Fourier basis or Chebyshev polynomials or other functions.

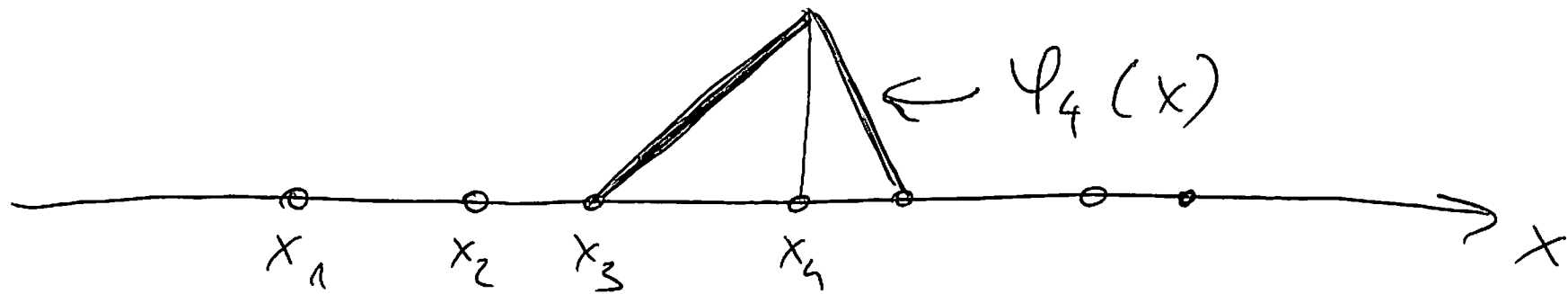
The key idea of the

## Finite Element Method

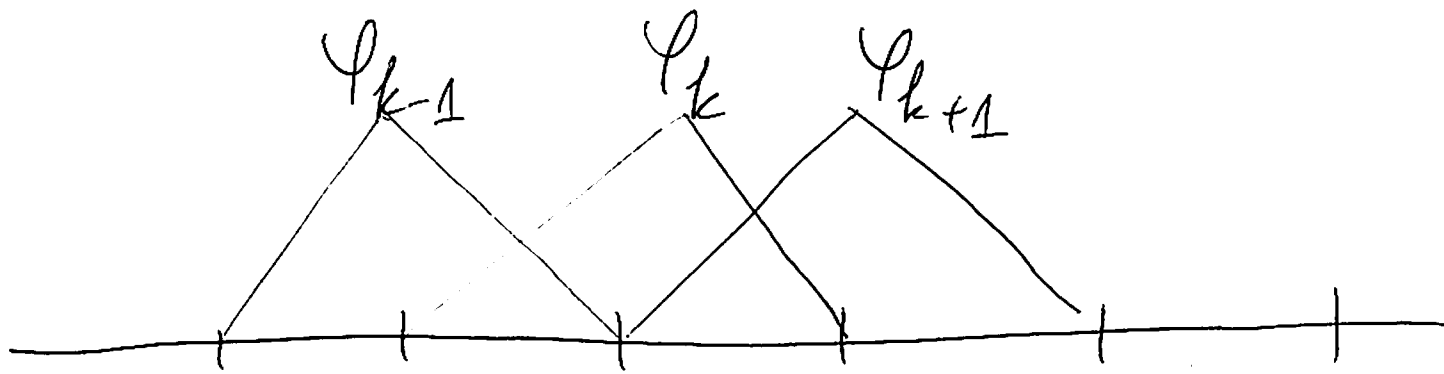
is to choose basis functions that are compactly supported over elements that tile the domain

That is, the basis functions are local rather than global. (12)  
This makes the stiffness matrix sparse

We need piecewise differentiable functions for our BVP, so choose linear basis "hat" functions:



$$x_{k+1} - x_k = h_k$$



Since the supports of  $\psi_{k-1}$  and  $\psi_{k+1}$  do not overlap, the stiffness matrix is tridiagonal

$\psi_k$  is called nodal basis function

$\delta_k = u_m(x_k)$  is the nodal value

$x_k$ 's are nodes

$$h_i = x_i - x_{i-1}$$

(Notation from section 0.5 in book  
by Susanne Brenner)

$$h = \max h_i$$

Take equation

$$-\frac{d^2 u}{dx^2} = f$$

$$\left\{ \begin{aligned} A_{i,i+1} &= \int_{x_i}^{x_{i+1}} \psi_i(x) \psi_{i+1}(x) dx = -\frac{1}{h_{i+1}} \end{aligned} \right.$$

$$A_{i,i} = \int_{x_{i-1}}^{x_{i+1}} \psi_i^2(x) dx = \frac{1}{h_i} + \frac{1}{h_{i+1}}$$

And

$$(f, \psi_i) = \int_{x_{i-1}}^{x_{i+1}} f(x) \psi_i(x) dx$$

Now we cannot compute this exactly for arbitrary  $f(x)$ .

In FEM one generally uses Gauss quadrature to approximate integrals. Here, to first order

$$(f, \psi_i) = \frac{1}{2} (h_i + h_{i+1}) (f(x_i) + O(h))$$

So the FEM Galerkin approach  
with linear elements gives the  
linear system

(16)

$$-\frac{2}{h_i + h_{i+1}} \left[ \frac{U_{i+1} - U_i}{h_{i+1}} - \frac{U_i - U_{i-1}}{h_i} \right] = f(x_i) + O(h)$$

LTE is only first order

But if  $h_i = h = \text{const}$  we get  
our usual second order

$$\frac{U_{i+1} - 2U_i + U_{i-1}}{h^2} = f(x_i)$$



It turns out that even  
for unequal  $h_i$  the FEM  
discretization is second-order accurate,  
even in the  $L_\infty$  norm! (17)

Proving this is greatly aided by the  
variational structure built into FEM

Theorem:

$$\|u - u_m\|_\infty \leq C h^2 \|u''\|_\infty$$

Proof uses Greens functions  
similar to Finite Difference proof...

Some error estimates:

(18)

Denote by  $S = \text{span} \{ \psi_1, \dots, \psi_m \}$

$$S \subset V = \{ \vartheta \in L^2(0,1) :$$

sufficiently smooth:  $\int |\vartheta'|^2 dx < \infty$  and  
(essential) BCs:  $\vartheta(0) = \vartheta(1) = 0$  }

Galerkin equations (Ritz-Galerkin)

Find

$$u_S \in S \quad \text{s.t.} \quad a(u_S, \vartheta) = (f, \vartheta) \\ \forall \vartheta \in S$$

where

$$a(u, v) = \int_0^1 u'(x) v'(x) dx$$

bilinear form

Defines a norm (energy norm):

$$\|v\|_E = \sqrt{a(v, v)}$$

Schwarz' inequality:

$$|a(u, v)| \leq \|u\|_E \|v\|_E$$

# Approximation error

result  
(Brenner book) (20)

$$\|u - u_S\|_E = \min \left\{ \|u - v\|_E : v \in S \right\}$$

i.e. the Galerkin approximation is the best one in the subspace  $S$  among all, in the energy norm

But what about the  $L_2$  norm?

$L_2$  is weaker than energy norm  
since no derivatives.

So we expect  $L_2$  to be even better

Assume the following  
approximation assumption

21

MUST  
CHECK!

$$\|u - u_s\|_2 \leq \epsilon \|u - u_s\|_E$$

where  $\epsilon$  is small, e.g.,  $\epsilon \sim h$  or  $\epsilon \sim h^2$

Then

$$\|u - u_s\|_2 \leq \epsilon \|u - u_s\|_E \leq \epsilon^2 \|u''\|$$

$$\Rightarrow \|u - u_s\|_2 \leq \epsilon^2 \|f\|$$

For FEM linear elements

$$\underline{\underline{\epsilon \sim h}}$$

What if we had Neumann conditions,

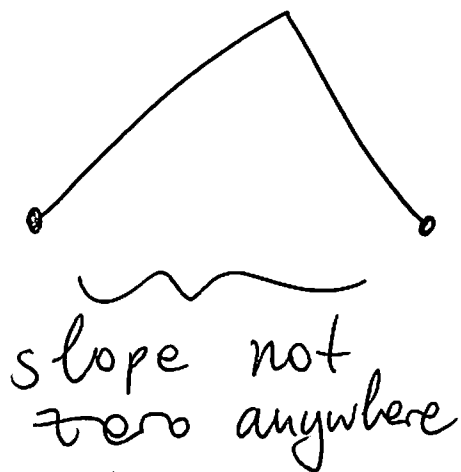
(22)

$$-u'' = f \quad \text{in } (0,1)$$

$$u(0)=0 \quad \text{and} \quad u'(1)=0$$

(\*)

It does not seem possible to impose  $u'(1)=0$  with the linear hat functions ! ? !



But, turns out we don't need to impose Neumann BC explicitly

Theorem (see 1D proof in  
theorem 0.1.4 in Brenner ch. 0): (23)

$\exists \neq u \in V$  and

$$a(u, v) = f(u, v) \quad \forall v \in V$$

where

$$V = \left\{ v \in L^2(0,1) : a(v, v) < \infty \text{ and } v(0) = 0 \right\}$$

only

Dirichlet BC

then  $u$  solves the PDE (\*)

the Dirichlet BC is essential  
the Neumann BC is natural

# Summary of FEM philosophy:

(24)

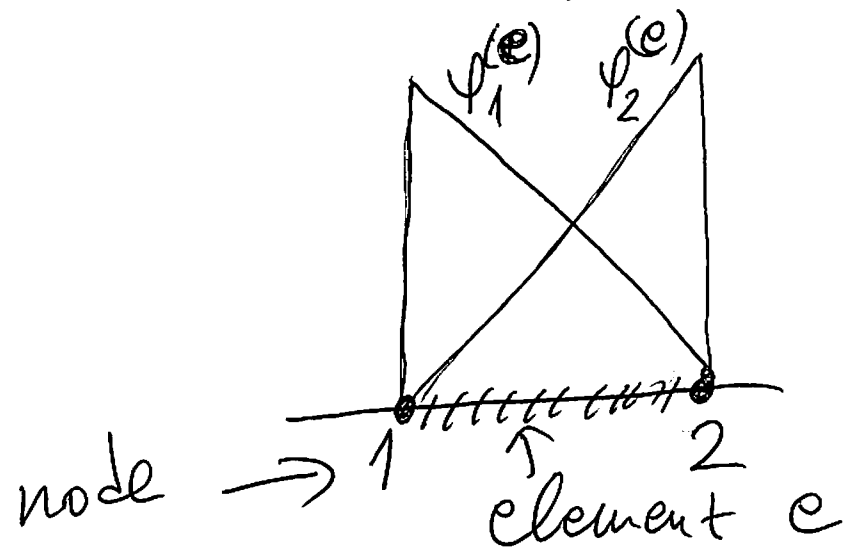
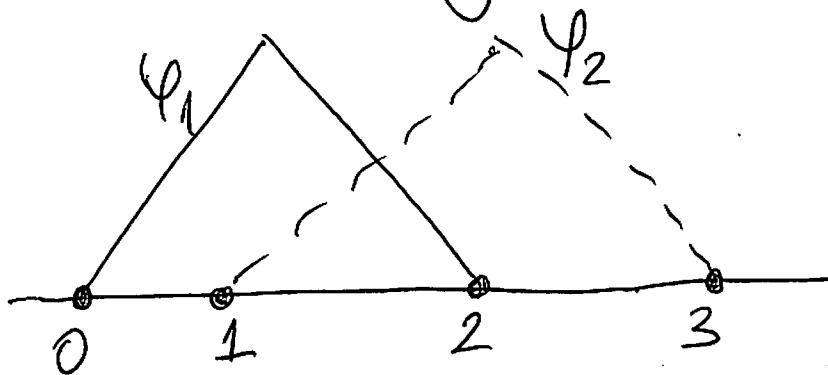
- ① Approximate solution in a finite-dimensional subspace  $V_0 + \overset{0}{H}_m \subset H$   
( $V_0 + S \subset V$ )
- ② Retain only essential BCs  
(variational formulation)
- ③ Make residual orthogonal to  $\overset{0}{H}_m$   
(Galerkin eqs.)
- ④ Integrate by parts to remove derivatives
- ⑤ Use compactly-supported basis (FEM)



# FEM in 2D

(25)

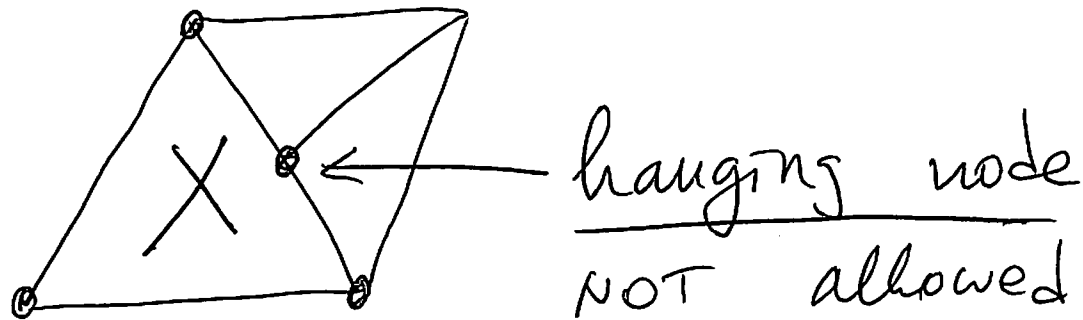
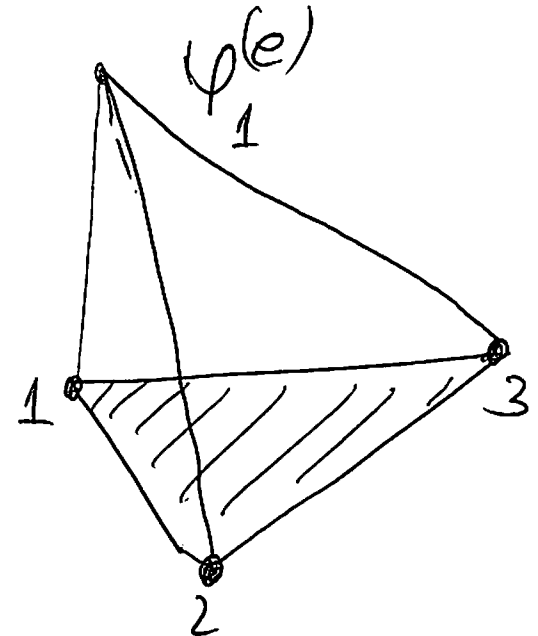
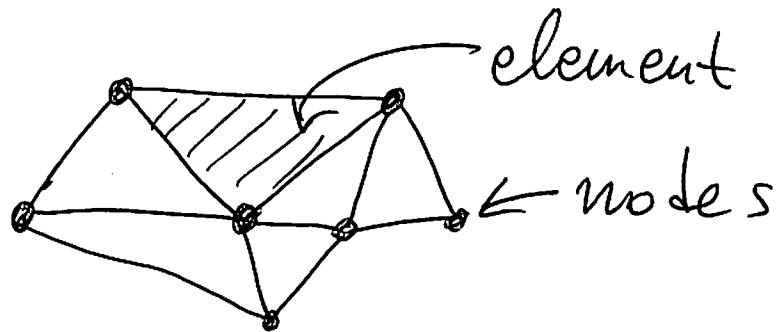
Instead of thinking of the basis functions being tent functions, associated with nodes of the grid, it is better to think of basis as being associated with elements



Now this is easy to transfer  
to two dimensions via

(26)

triangular elements



Every triangle has 3 vertices/nodes

Basis functions are now  
tent functions over each triangle

(27)

$$\varphi_i(e) \leftarrow \text{element} = \begin{cases} 1 & \text{at vertex } i \\ 0 & \text{at all other vertices} \end{cases}$$

$i \swarrow$   
vertex of element

$$\varphi_i = \underbrace{\alpha + \beta x + \gamma y}$$

three coefficients (=)  
three values at vertices of  
triangle

Linear interpolant (piecewise linear)  
is unique given vertex values.

The unknowns are now the values at the vertices

(28)

$$u = \sum_e \sum_{i \in e} u_i \psi_i^{(e)}(x, y)$$

One vertex (degree of freedom) is shared among multiple triangles.

Consider the Poisson equation.

Stiffness matrix is

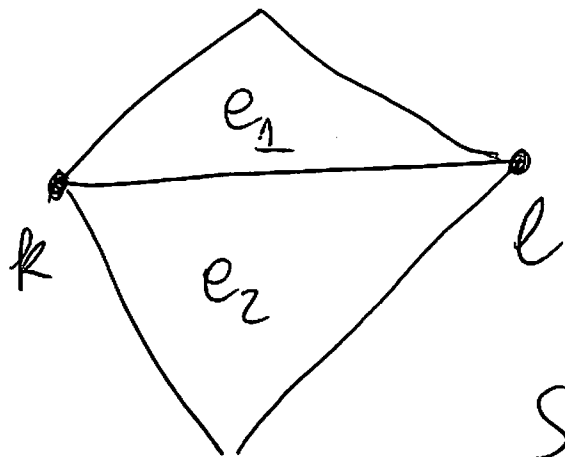
$$a_{k,l} = \int_{\Omega} (\vec{\nabla} \psi_k) \cdot (\vec{\nabla} \psi_l) dA$$

vertices

$$a_{k,l} = \sum_e \int \left( \frac{\partial \psi_k}{\partial x} \frac{\partial \psi_l}{\partial x} + \frac{\partial \psi_k}{\partial y} \frac{\partial \psi_l}{\partial y} \right) dA$$

$\sqrt{e}$   
triangle

But integrand is only nonzero if vertices  $k$  and  $l$  belong to triangle  $e$ , i.e., if they share triangle  $e$ .



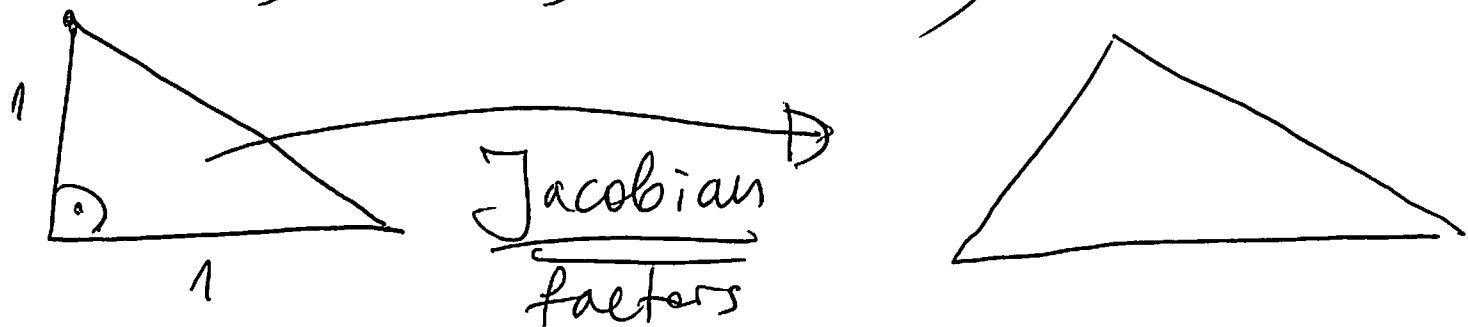
$$a_{k,l} = \sum_{(k,l) \in e} a_{k,l}^{(e)}$$

Stiffness matrix assembly

We just need to worry about one element at a time. (30)

Inside one element the functions are linear so the integrals are easy to compute analytically.

Usually this is precomputed on a reference triangle that is mapped by an affine transformation (translate, rotate, stretch)

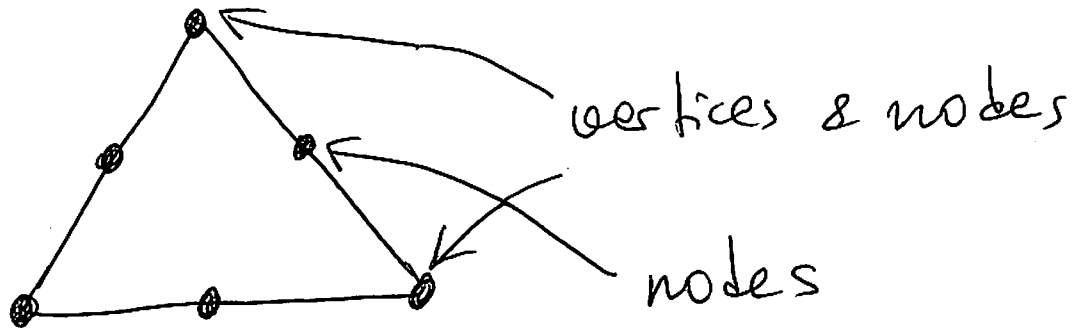


(see some examples / images)  
on webpage

31

If we want to do quadratic interpolation on triangles, then inside each triangle

$$\psi(x,y) = \underbrace{\alpha + \beta x + \gamma y + \delta x^2 + \eta xy + \zeta y^2}_{\text{Six coefficients} \Rightarrow \text{six nodes}}$$

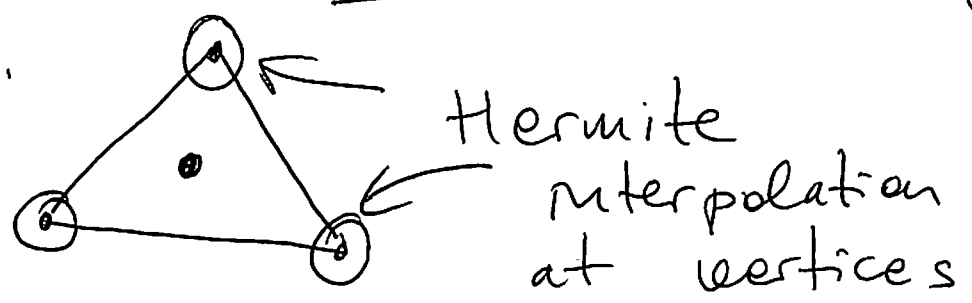


Observe now that along each edge the interpolant is a parabola, which is uniquely defined by the tree nodal values on each edge, so the interpolant is continuous across triangle boundaries.

This is all we need for Poisson

But interpolant is not continuously differentiable.

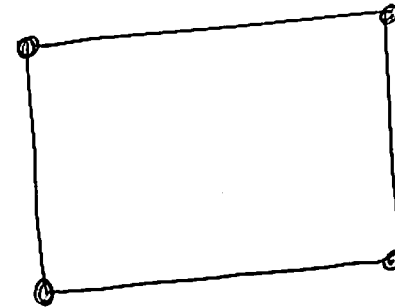
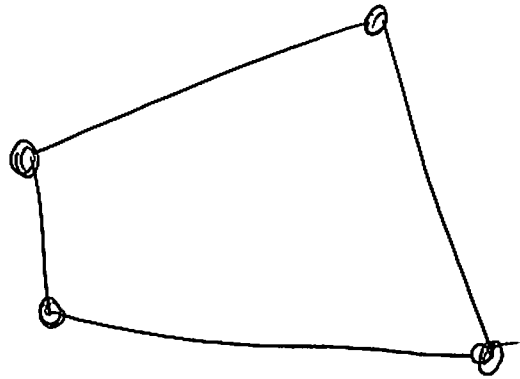
Instead, do





It is possible sometimes to use quadrilateral meshes

(33)



reference square

Now we use bilinear interpolation, i.e., inside each element (reference element)

$$\psi(x, y) = \psi(x) \psi(y) \leftarrow \text{continuous at edges}$$

or

$$\underbrace{\alpha + \beta x + \gamma y + \delta xy}_{\text{four coefficients}}$$

There is a zoo of FEM (34)  
basis functions on triangular,  
tetrahedral, rectangular, hexagonal,  
and other grids.

Understanding which element  
is required (how much smoothness is  
needed, conforming vs. non-conforming)  
and obtaining error estimates in  
various Sobolev norms is the  
topic of FEM textbooks and  
not trivial.