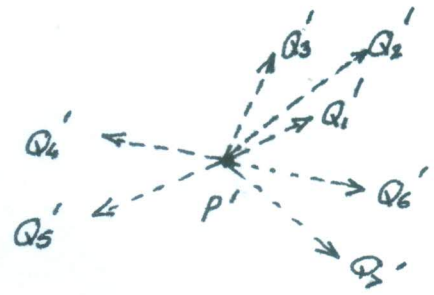
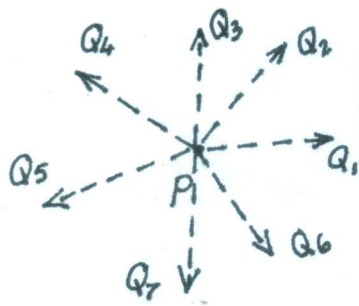


# Consequences of strain derivation

T-1



Let  $\vec{PQ_i} = \vec{\Delta X}^{(i)}$

and  $\vec{P'Q'_i} = \vec{\Delta X}'^{(i)}$ , then

with  $l_0^{(i)} = |\vec{\Delta X}^{(i)}|$ ;  $l^{(i)} = |\vec{\Delta X}'^{(i)}|$ , we have

$$\frac{\Delta l^{(i)}}{l_0^{(i)}} = \frac{l^{(i)} - l_0^{(i)}}{l_0^{(i)}} = \frac{\vec{\Delta X}^{(i)T} [\epsilon] \vec{\Delta X}^{(i)}}{l_0^{(i)2}}$$

where  $[\epsilon]_P = \begin{bmatrix} \epsilon_{11} & \epsilon_{12} & \epsilon_{13} \\ \epsilon_{12} & \epsilon_{22} & \epsilon_{23} \\ \epsilon_{13} & \epsilon_{23} & \epsilon_{33} \end{bmatrix}_{\text{at } P}$

as the strain tensor at  $P$ .

- Knowing  $[\epsilon]_P$  we can get stretch of ANY infinitesimal line element  $\vec{PQ_i}$  at  $P$ .

E.g.  $\vec{\Delta X}^{(1)} = \Delta x \hat{i} \Rightarrow \frac{\Delta l^{(1)}}{l_0^{(1)}} = \frac{\{\Delta x \ 0 \ 0\} \begin{Bmatrix} \epsilon_{11} \Delta x \\ \epsilon_{21} \Delta x \\ \epsilon_{31} \Delta x \end{Bmatrix}}{\Delta x^2}$

$$= \frac{\epsilon_{11} \cdot \Delta x^2}{\Delta x^2} = \epsilon_{11}$$

$\vec{\Delta X}^{(2)} = \frac{\Delta x}{\sqrt{2}} (\hat{i} + \hat{j}) \Rightarrow \frac{\Delta l^{(2)}}{l_0^{(2)}} = \frac{\Delta x^2 \{1 \ 1 \ 0\} \begin{Bmatrix} \epsilon_{11} + \epsilon_{12} \\ \epsilon_{21} + \epsilon_{22} \\ \epsilon_{31} + \epsilon_{32} \end{Bmatrix} \times \frac{1}{2}}{\Delta x^2}$

$$= \boxed{\epsilon_{11} + 2\epsilon_{12} + \epsilon_{22}} / 2$$

and so on!!

Eigenvalue problem for strain tensor:

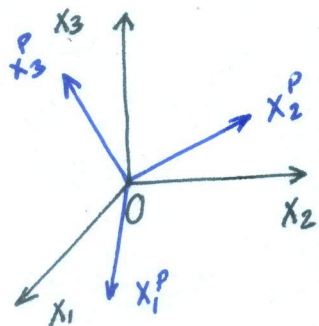
$$[E] \vec{n} = \lambda \vec{n} \Rightarrow \text{we get eigenvalues } \lambda_1, \lambda_2, \lambda_3$$

and corresponding eigenvectors  $\vec{n}^{(1)}, \vec{n}^{(2)}, \vec{n}^{(3)}$ .

— Since  $[E]$  is symmetric,  $\{\lambda_i\}_{i=1}^3$  is real and

$$\vec{n}^{(1)} \perp \vec{n}^{(2)} \perp \vec{n}^{(3)} \Leftrightarrow \text{form an orthogonal coordinate system}$$

principal coordinates / axes.  $\rightarrow x_1^p - x_2^p - x_3^p$



$$\text{Now } [R] = [\vec{n}^{(1)} \quad \vec{n}^{(2)} \quad \vec{n}^{(3)}]$$

$$\Rightarrow [E][R] = [\lambda_1 \vec{n}^{(1)} \quad \lambda_2 \vec{n}^{(2)} \quad \lambda_3 \vec{n}^{(3)}]$$

$$= [n^{(1)} \quad n^{(2)} \quad n^{(3)}] \begin{bmatrix} \lambda_1 & 0 & 0 \\ 0 & \lambda_2 & 0 \\ 0 & 0 & \lambda_3 \end{bmatrix}$$

$$\Rightarrow [E][R] = [R][\Lambda] \Rightarrow [E] = [R][\Lambda][R]^T$$

\* Note that  $\{\Delta x^p\} = [R]^T \{\Delta x\} = [Q] \{\Delta x\}$  with

$$[Q] = [R]^T \Rightarrow [E] = [Q]^T [\Lambda] [Q]$$

$$\text{or } \boxed{[\Lambda] = [Q][E][Q]^T}$$

\* Note that  $\vec{\Delta x'} = \Delta x \vec{n}^{(i)}$  gives  $\frac{\Delta l}{l_0} = \boxed{\lambda_i}$

\* No shear in principal strain coordinate system.

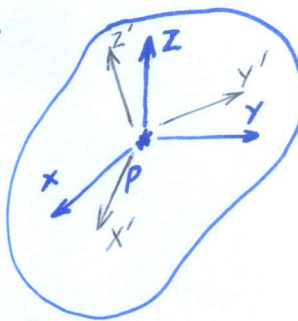
## Stress and strain transformation

- Let point P have coordinates  $(x, y, z)$

- Let  $(x', y', z')$  be coordinates in a new, rotated coordinate system obtained by:

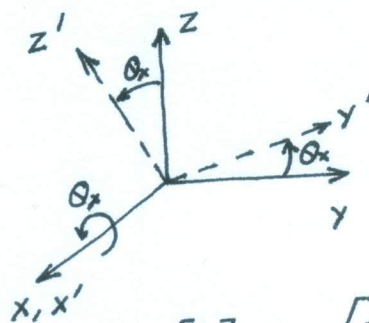
$$\begin{Bmatrix} x' \\ y' \\ z' \end{Bmatrix} = [R] \begin{Bmatrix} x \\ y \\ z \end{Bmatrix}$$

rotation matrix



Examples of  $[R]$ :

$$[R] = \begin{bmatrix} \cos \theta_z & \sin \theta_z & 0 \\ -\sin \theta_z & \cos \theta_z & 0 \\ 0 & 0 & 1 \end{bmatrix}$$



$$[R] = \begin{bmatrix} 1 & 0 & 0 \\ 0 & \cos \theta_x & \sin \theta_x \\ 0 & -\sin \theta_x & \cos \theta_x \end{bmatrix}$$

$$[\sigma'] = [R][\sigma][R]^T \quad ; \quad [\epsilon'] = [R][\epsilon][R]^T$$

Principal stress directions:

$$[\sigma]\{\vec{n}\} = \lambda \{\vec{n}\} \Rightarrow \text{Eigenvalues (principal stresses)} \\ \lambda_1, \lambda_2, \lambda_3$$

Principal directions (or eigenvectors)  $\vec{n}^1, \vec{n}^2, \vec{n}^3$  with  $|\vec{n}^1| = |\vec{n}^2| = |\vec{n}^3| = 1$   
and  $\vec{n}^1 \perp \vec{n}^2 \perp \vec{n}^3$ .

$$\begin{Bmatrix} \vec{n}^1 \\ \vec{n}^2 \\ \vec{n}^3 \end{Bmatrix} = \begin{bmatrix} n_1^1 & n_1^2 & n_1^3 \\ n_2^1 & n_2^2 & n_2^3 \\ n_3^1 & n_3^2 & n_3^3 \end{bmatrix} \begin{Bmatrix} \hat{i} \\ \hat{j} \\ \hat{k} \end{Bmatrix} \Rightarrow \{\vec{n}\} = \underbrace{[N]}_{[R]^T} \{\hat{i}\} \\ \Rightarrow \boxed{\{x^p\} = [R^p]\{x\}}$$



Similarly for  $[\epsilon]$  and  $[\epsilon^p] = [\lambda^p] \leftarrow$  diagonal  
 principal strains

If the state of stress at a point P is :

$$[\sigma] = \begin{bmatrix} 200 & -100 & 0 \\ -100 & 200 & 0 \\ 0 & 0 & 0 \end{bmatrix} \text{ MPa.} \quad \leftarrow \sim \text{ PLANE STRESS}$$

$$\Rightarrow [\sigma] \{n\} = \lambda \{n\} \Rightarrow \det ([\sigma] - \lambda [I]) = 0$$

$$\Rightarrow \left| \begin{bmatrix} 200-\lambda & -100 & 0 \\ -100 & 200-\lambda & 0 \\ 0 & 0 & -\lambda \end{bmatrix} \right| = 0 \Rightarrow (200-\lambda)(200-\lambda)(-\lambda) - 100 \times [0 - (-100)(-\lambda)] = 0$$

$$\Rightarrow -\lambda [(200-\lambda)^2 - 100^2] = 0$$

$$\Rightarrow \lambda = 0, 100, 300 \text{ MPa}$$

For  $\lambda = 0$

$$\begin{bmatrix} 200 & -100 & 0 \\ -100 & 200 & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{Bmatrix} n_1' \\ n_2' \\ n_3' \end{Bmatrix} = \vec{0} \Rightarrow 200 n_1' = 100 n_2'$$

$$\Rightarrow n_1' = \frac{1}{2} n_2'$$

$$0 n_3' = 0 \Rightarrow n_3' \text{ is anything}$$

$$100 n_1' = 200 n_2' \Rightarrow n_1' = n_2' = 0$$

$$\text{or } \vec{n}_1 = \begin{Bmatrix} 0 \\ 0 \\ 1 \end{Bmatrix}$$

For  $\lambda = 100$

$$\begin{bmatrix} 100 & -100 & 0 \\ -100 & 100 & 0 \\ 0 & 0 & -100 \end{bmatrix} \begin{Bmatrix} n_1^2 \\ n_2^2 \\ n_3^2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \end{Bmatrix} \Rightarrow \begin{matrix} n_1^2 = n_2^2 \\ = \frac{1}{\sqrt{2}} \\ n_3^2 = 0 \end{matrix}$$

$$\Rightarrow \vec{n}_2 = \begin{Bmatrix} \frac{1}{\sqrt{2}} \\ \frac{1}{\sqrt{2}} \\ 0 \end{Bmatrix}$$

$$\vec{n}_3 = \begin{Bmatrix} \frac{1}{\sqrt{2}} \\ -\frac{1}{\sqrt{2}} \\ 0 \end{Bmatrix}$$

$$\lambda = 200 \Rightarrow \begin{bmatrix} -100 & -100 & 0 \\ -100 & -100 & 0 \\ 0 & 0 & -200 \end{bmatrix} \begin{Bmatrix} n_1^3 \\ n_2^3 \\ n_3^3 \end{Bmatrix} = \vec{0}$$

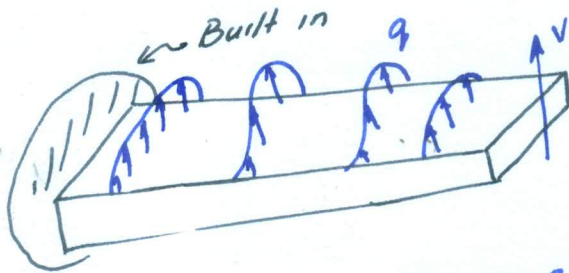
If  $\vec{n}_2 = \hat{i}_p$ ;  $\vec{n}_3 = \hat{j}_p$ ;  $\vec{n}_{01} = \vec{k}_p$  then

$$\begin{Bmatrix} \hat{i}_p \\ \hat{j}_p \\ \hat{k}_p \end{Bmatrix} = \underbrace{\begin{bmatrix} \frac{1}{\sqrt{2}} & \frac{1}{\sqrt{2}} & 0 \\ \frac{1}{\sqrt{2}} & -\frac{1}{\sqrt{2}} & 0 \\ 0 & 0 & 1 \end{bmatrix}}_{[R^p]} \begin{Bmatrix} \hat{i} \\ \hat{j} \\ \hat{k} \end{Bmatrix}$$

CHECK !

$$\begin{bmatrix} 100 & 0 & 0 \\ 0 & 200 & 0 \\ 0 & 0 & 0 \end{bmatrix} = [R^p][\sigma][R^p]^T$$

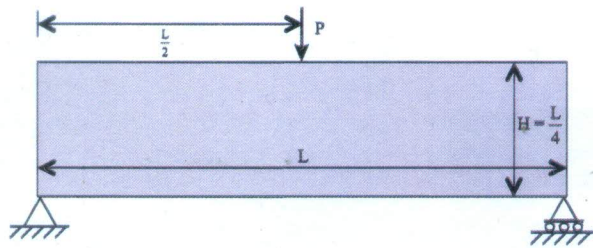
What is the scenario for an actual structure ?



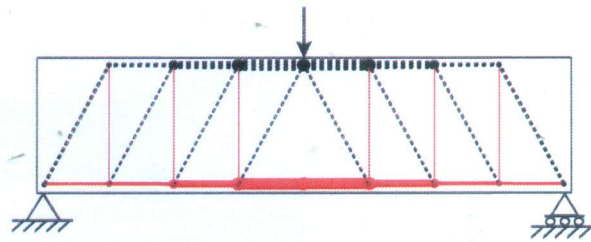
← If the structure is subjected to these loads then as a designer I have

to figure out → profile / dimensions; material used in order to get a safe structure.

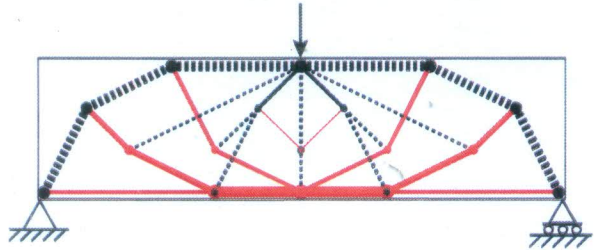
— For this we need to obtain stress at every point and then CALCULATE  $\bar{\sigma}_{RMS}$  or  $T_{max}$ ; CHECK if point is yielding; If yes, reinforce material to reduce local stress or change material to ensure no yielding; If no, then remove USELESS material.



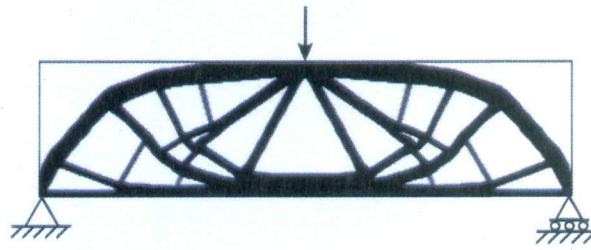
(a) Design domain



(b) Traditional strut-and-tie model



(c) Optimized truss model



(d) Optimized continuum model

→ Figure (a)  $\Rightarrow$  initial block of material

→ Figure (d)  $\Rightarrow$  optimized structure for this load  
i.e. point load  $P$  at top.