

APPENDIX B
SIZING COMPONENTS

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HX	Heat Exchanger	B11
INIT	Initialization	B12
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LINE	Line	B14
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TURB	Turbine	B19
VALVE	Valve	B20
VCØMP	Vapor Cycle Compressor	B21
WSEP	Water Separator	B22

- Notes: 1. Data card fields which refer to the parameter table are indicated by brackets, [].
2. A card column specified by a single value represents the righthand column of a four column integer data field. Data values punched in these fields must be right justified.

B1

Sizing Component APU

Auxiliary Power Unit

Purpose: This component will compute weight, cost units, reliability index, development risk factor, fuel flow rate, and volume for APU. The available APU types are:

IAPUT	Type
1	bleed air extraction with or without shaft power extraction
2	shaft power extraction only

Equations:

1. Weight:

For IAPUT = 1:

$$Wt = \frac{0.398 (EHP)_{ECS}}{(4.215 \times 10^{-4})(EHP)_T + 0.159}$$

For IAPUT = 2:

$$Wt = \frac{0.398 (EHP)_{ECS}}{(6.29 \times 10^{-4})(EHP)_T + 0.253}$$

where

$$(EHP)_{ECS} = (HP)_{\text{shaft}} + W C_p T_{\text{out}} [1 - (\frac{P_{\text{out}}}{P_{\text{in}}})^{\frac{1-\gamma}{\gamma}}] / 42.42$$

$$(EHP)_T = (EHP)_{ECS} + (HP)_{\text{other system}}$$

2. Cost Units:

$$CU = 7.3(EHP)_{ECS}$$

3. Reliability Index:

$$RI = 0.24971$$

4. Development Risk Factor:

$$DR = 1.15$$

5. Weight Standard Error:

For IAPUT = 1:

$$Wt_d = 0.1809 Wt$$

For IAPUT = 2:

$$Wt_d = 0.3241 Wt$$

6. Installation Weight Factor:

$$Wt_i = 1.5126 Wt$$

(B1.1)(APU)

7. Fuel Flow Rate:

For IAPUT = 1:

$$W_{fu} = [1.5 - 1.158 \times 10^{-3} (EHP)_T] (EHP)_{ECS}$$

For IAPUT = 2:

$$W_{fu} = [1.55 - 2.43 \times 10^{-3} (EHP)_T] (EHP)_{ECS}$$

8. Volume:

For IAPUT = 1:

$$V = \frac{(EHP)_{ECS}}{(1.343 \times 10^{-5}) (EHP)_T + 2.95 \times 10^{-3}}$$

For IAPUT = 2:

$$V = \frac{(EHP)_{ECS}}{(9.919 \times 10^{-6}) (EHP)_T + 6.83 \times 10^{-3}}$$

Restrictions:

1. This component can only be used for fluid type 2.

Notes:

1. The component printout consists of weight, cost units, reliability index, development risk factor, and the following items:

W - flow rate

PI - inlet pressure

P \emptyset - outlet pressure

TI - inlet temperature

T \emptyset - outlet temperature

HI - inlet humidity

H \emptyset - outlet humidity

FT - fluid type

FN - fluid properties table
relative number

SHAFT - shaft number

SHP - shaft horsepower

V \emptyset L - volume

WTBT - total bare weight

} for APU with bleed air
extraction by ECS

} for APU with shaft power
extraction by ECS

EHPT - total equivalent horsepower

FC - fuel flow rate (lb/hr)

2. The value of volume is stored in general argument 71.

Sizing Component APU

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-3	CN	APU component name
8	CC	change code
12	NC	card number
16	NL	leg number = 0 without bleed air extraction by ECS > 0 with bleed air extraction by ECS
20	NSI	inlet station number (not required if NL = 0)
24	NSØ	outlet station number (not required if NL = 0)
28	NST	shaft number = 0 without shaft power extraction by ECS > 0 with shaft power extraction by ECS
32	[WTM]	weight multiplier (optional)
36	[CUM]	cost unit multiplier (optional)
40	[RI]	reliability index (optional)
44	[DRM]	development risk factor multiplier (optional)
48	IAPUT	APU type = 1 bleed air extraction with or without shaft power extraction = 2 shaft power extraction only
52	[ØEHP]	equivalent horsepower required by other systems

SECTION XIX

B2 - REPLACEMENT SIZING COMPONENT BOIL

Boiler

Purpose: This component will compute core dimensions, weight, cost units, reliability index, development risk factor, core volume, and the required expendable coolant weight for boilers. The available types of boilers are:

IBT Type

- 1 integral boiler
- 2 boiler with separate storage tank

Equations:

1. Weight:

(a) Core Weight:

The same equations for heat exchanger core weight are applicable to the boiler core. (See HX component)

(b) Boiler Weight (dry):

For IBT = 1:

$$Wt = Wt_{core} + 3.4722 \times 10^{-4} \rho_{metal} (V + V_{liquid})^{2/3}$$

for IBT = 2:

$$Wt = Wt_{core} + 3.0382 \times 10^{-4} \rho_{metal} V_{liquid}^{2/3}$$

2. Cost Units:

$$CU = 29.7 + 2.72 Wt$$

3. Reliability Index:

$$RI = 0.02165$$

4. Development Risk Factor:

If the average pressure is less than or equal to 314.7 psi:

$$DR = 1.15$$

If the average pressure is greater than 314.7 psi:

$$DR = 1.15 [1.0 + (\frac{P_{av} - 314.7}{283.0})^2]$$

5. Weight Standard Error:

$$Wt_d = 0.12 Wt$$

6. Installation Weight Factor:

$$W_{t_i} = 0.205 \text{ Wt}$$

7. Core dimensions (if not specified):

Core dimensions L_H , L_C , and L_N are obtained iteratively by the generalized Newton Raphson method from the following three equations:

$$L_N = r_a L_C$$

$$\Delta P_H = \frac{Re_H^2 \mu_H^2 (1 + \psi_H^2)}{1029.568 r_{hH}^2 \rho_{1H}} \left[\frac{f_H L_H N_{PH} (\rho_{1H}/\rho_{2H} + 1)}{2r_{hH}(1 + \psi_H^2)} + \frac{\rho_{1H}}{\rho_{2H}} - 1 \right]$$

$$\epsilon = 1 - e^{-NTU}$$

where

$$r_a = \text{aspect ratio } (L_N/L_C)$$

$$= 1.0$$

$$NTU = \frac{\eta_{OH} j_H L_H N_{PH}}{r_{hH} Pr_H^{2/3}}$$

$$\psi_H = \frac{b_H \beta_H r_{hH} N_H}{(b_H + b_C + 2 t_{SP}) N_H + b_C + 2 t_{SP}}$$

$$Re_H = \frac{0.8 r_{hH} W_H N_{PH}}{\mu_H L_C L_N \psi_H}$$

$$\eta_{OH} = 1 - (A_F/A)_H \left[1 - \frac{\tanh(m_H \ell_{eH})}{m_H \ell_{eH}} \right]$$

$$m_H = \left[\frac{1800 C'_{pH} \mu_H Re_H j_H (1 + t_{FH}/\xi_H)}{r_{hH} t_{FH} Pr_H^{2/3} k_H} \right]^{1/2}$$

$$SHR = (T_{in} - \bar{T}_{out})/(T_{in} - T_{out})$$

$$C'_{pH} = C_{pH} \cdot SHR$$

8. Core Volume:

$$V = L_H L_C L_N$$

9. Cold Side Liquid Volume:

$$V_{liquid} = 1728 W C_p (T_{in} - T_{out}) / (H_{fg} \rho_{sat})^\tau$$

$\psi = \frac{\text{free flow area}}{\text{frontal area}}$

10. Volume of hot side liquid inside the core:

$$V_{LH} = \psi_H V$$

Required Tables:

- Table ID (IFH, 31 and IFGC, 31) - fin geometry (hot side and cold side), refer to HX component for table description.
- f - Table ID (IFH, 32) - fin friction factor (hot side), refer to HX component for table description.
- j - Table ID (IFH, 33) - fin Colburn modulus (hot side), refer to HX component for table description.
- t_{SP} - Table ID (ITSP, 34) - separation plate thickness, refer to HX component for table description.
- Table ID (IMH, 20) - hot side fin material properties.
- Table ID (IMC, 20) - cold side fin material properties.
- Table ID (IMSP, 20) - separation plate material properties.
- Table ID (IMW, 20) - storage tank wall material properties.

Restrictions:

1. This component is used for sizing plate fin boilers.
2. This component can only be used for fluid types 1 and 2 on the hot side.
3. The core dimensions L_H , L_C , L_N must either be all specified by the user or be all unspecified.

Notes:

1. Two consecutive data cards are required, BØILL and BØIL2. The BØILL card must precede the BØIL2 card.
2. The same table relative number is used for the fin geometry table, the fin friction factor table, and the fin Colburn modulus table of the hot side.
3. Hot side fluid properties are evaluated at average temperature, $T_{ave} = (T_{in} + T_{out})/2$, and average pressure, $P_{ave} = (P_{in} + P_{out})/2$.
4. The component printout consists of boiler weight, cost units, reliability index, development risk factor, and the following items:
 - W - flow rate
 - PI - inlet pressure
 - PØ - outlet pressure

{ hot side

TI - inlet temperature
 T_O - outlet temperature
 HI - inlet humidity
 H_O - outlet humidity
 FT - fluid type
 FN - fluid properties table relative number
 LH - hot side flow length
 LC - cold side flow length
 LN - no flow length
 V_{OL} - core volume
 W_{TC} - core weight
 EWT - expendable coolant weight
 EFF - effectiveness (not printed if core dimensions are specified)
 NTU - number of heat transfer units (not printed if core dimensions are specified)
 VL - volume of hot side liquid inside the core (not printed if hot side is a gas)

hot side

5. Four performance map options are available. They are
 - Option 0 - no maps
 - Option 1 - print maps
 - Option 2 - punch maps
 - Option 3 - print and punch maps
 The map produced will consist of effectiveness versus hot side flow rate, and $\sigma\Delta P$ (if a gas) or ΔP (if a liquid) of hot side versus hot side flow rate.
6. If core dimensions are not specified by the user and the iteration dump option is selected, the following information is printed:
 - (a) State Variables (S.V.) - the two values printed are L_H and L_C , respectively.
 - (b) Error Variables (E.V.) - the two values printed are the difference of the calculated and the given hot side pressure drops, and the difference of the calculated and the given effectiveness, respectively.

- (c) The four values printed after the error variables are Re_H , NTU, ϵ , and η_{OH} , respectively.
7. The convergence limits for pressure drop and effectiveness are 0.01 and 0.0001, respectively. The upper and the lower limits for the state variables are 100.0 and 0.1, respectively.
8. Values of L_H , L_C , L_N , V , and V_{LH} are stored in general arguments 71, 72, 73, 74, and 75, respectively.

Sizing Component BØIL

Data Card Format

1. BØILL Data Card

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-5	CN	BØILL component name
8	CC	change code
12	NC	card number
16	NL	leg number, hot side
20	NSI	inlet station number, hot side
24	NSØ	outlet station number, hot side
28	[WTM]	weight multiplier (optional)
32	[CUM]	cost unit multiplier (optional)
36	[RI]	reliability index (optional)
40	[DRM]	development risk factor multiplier (optional)
44	[BT]	boiling temperature of cold side liquid
48	[DENS]	cold side liquid density at boiling temperature
52	[HFG]	heat of vaporization of cold side liquid
56	[TIME]	boiler usage time
60	[WTE]	extra cold side liquid weight
64	IBT	boiler type = 1 integral boiler = 2 boiler with separate storage tank
68	MAP	performance map options = 0 no maps = 1 print maps = 2 punch maps = 3 print and punch maps

2. BØIL2 Data Card

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-5	CN	BØIL2 component name
8	CC	change code
12	NC	card number
16	[LH]	core flow length, hot side = 0 computed > 0 user specified
20	[LC]	core flow length, cold side = 0 computed > 0 user specified
24	[LN]	no flow length = 0 computed > 0 user specified
28	NPH	number of passes, hot side
32	IMH	fin material properties table relative number, hot side (type 20)
36	IMC	fin material properties table relative number, cold side (type 20)
40	IFH	fin variable tables relative number, hot side (types 31, 32, 33)
44	IFGC	fin geometry table relative number, cold side (type 31)
48	IMSP	separation plate material properties table rela- tive number (type 20)
52	ITSP	separation plate thickness table relative number (type 34)
56	IMW	storage tank wall material properties table relative number (type 20)

Sizing Component CNTRL

System Control

Purpose: This component will compute weight, cost units, reliability index, development risk factor, and volume for system control. The available control types are:

ICT Type

- 1 temperature control, electronic, fighter or light bomber
- 2 temperature control, electronic, cargo or heavy bomber
- 3 temperature control, pneumatic or electro-pneumatic
- 4 cabin pressure control, fighter or light bomber
- 5 cabin pressure control, cargo or heavy bomber

Equations:

1. Weight:

For ICT = 1 & 2:

$$Wt = \left[\frac{NIA\emptyset}{3} + Wt_{\text{selector}} + 0.2 \text{ (number of sensors)} \right] K_T K_P$$

For ICT = 3:

$$Wt = (0.5D + 1.75) K_T K_P$$

For ICT = 4:

$$Wt = 6.0$$

For ICT = 5:

$$Wt = 0.096W + 51.45$$

2. Cost Units:

For ICT = 1:

$$CU = [14.6 + 0.545 (NIA\emptyset)] Wt K_T K_P$$

For ICT = 2:

$$CU = [14.85 + 1.04 (NIA\emptyset)] Wt K_T K_P$$

For ICT = 3:

$$CU = (2.48D + 13.6) K_T K_P$$

For ICT = 4:

$$CU = 49.5$$

For ICT = 5:

$$CU = (6.45 + 0.0176W) Wt$$

3. Reliability Index:

For ICT = 1 & 2:

$$RI = 0.04628 + 0.03781 \text{ (number of sensors)}$$

For ICT = 3:

$$RI = 0.00582 + 0.03781 \text{ (number of sensors)}$$

For ICT = 4 & 5:

$$RI = 0.24448$$

4. Development Risk Factor:

$$DR = 1.0$$

5. Weight Standard Error:

For ICT = 1, 2, and 3:

$$W_{t_d} = 0.41 W_t$$

For ICT = 4 & 5:

$$W_{t_d} = 0.24 W_t$$

6. Installation Weight Factor:

$$W_{t_i} = 0.205 W_t$$

7. Volume:

For ICT = 1 & 2:

$$V = [11.43 (\text{NIAØ}) + 25.71] K_T K_P$$

For ICT = 3:

$$V = (40.0D + 20.0) K_T K_P$$

For ICT = 4:

$$V = 250.0$$

For ICT = 5:

$$V = 8.065W + 5671.0$$

8. Thrust Recovery (for ICT = 5):

$$\text{Thrust} = 6.587 \times 10^{-4} W_j c \left[\frac{\frac{P_{\text{cabin}}}{P_{\text{ambient}}}}{\gamma - 1} \right]^{\frac{\gamma - 1}{\gamma}}$$

where

c = the speed of sound at the nozzle exit

$$W_j = W - \frac{31.82(C_A) P_{\text{cabin}}}{\sqrt{T_{\text{cabin}}}}$$

Restrictions:

1. This component can only be used for fluid type 2 if ICT = 5

Notes:

1. Temperature control sizing method applies to water separator anti-ice control, vapor cycle control, and other controls which modulate flow.
2. For ICT = 5, this component has an option to compute the thrust recovery.
3. The component printout consists of weight, cost units, reliability index, development risk factor, and the following items:

VOL - volume

ICT - control type

THRC - thrust recovery (for ICT = 5 and if thrust recovery option is used only)

Sizing Component CNTRL

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-5	CN	CNTRL component name
8	CC	change code
12	NC	card number
16	[WTM]	weight multiplier (optional)
20	[CUM]	cost unit multiplier (optional)
24	[RI]	reliability index (optional)
28	[DRM]	development risk factor multiplier (optional)
32	ICT	control type = 1 temperature control, electronic, fighter or light bomber = 2 temperature control, electronic, cargo or heavy bomber = 3 temperature control, pneumatic or electro-pneumatic = 4 cabin pressure control, fighter or light bomber = 5 cabin pressure control, cargo or heavy bomber
if ICT = 1 & 2		
36	[KT]	technical weighing factor
40	[KP]	requirements weighing factor
44	[WTS]	selector weight
48	NSR	number of sensors
52	NIAO	number of inputs and outputs
if ICT = 3		
36	[KT]	technical weighing factor
40	[KP]	requirements weighing factor
44	[D]	control valve diameter
if ICT = 4		
no additional data required.		
if ICT = 5		
36	NL	cabin leg number

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
40	NS	cabin outlet station number = 0 no thrust recovery > 0 thrust recovery computed
44	[CDA]	effective leakage area (not required if NS = 0)

Sizing Component COMP
Compressor

Purpose: This component will compute weight, cost units, reliability index, development risk factor, volume, wheel diameter and/or shaft speed for an air cycle compressor (centrifugal or axial). It will also compute weight, cost units, volume, efficiency, and input horsepower for drives. The available types of drives are:

IDT	Type
0	drive not to be sized
1	turbine
2	electric motor, AC
3	electric motor, DC
4	hydraulic motor
5	shaft, engine, or APU

Equations:

1. Weight:

For compressor:

$$Wt_{comp} = 0.4D^2$$

For drive (IDT = 2):

$$Wt_{drive} = 2.0 + 2.3 \times 10^5 \frac{HP^{5/6}}{N_{drive}^{1.25}}$$

For drive (IDT = 3):

$$Wt_{drive} = 1.5 + 3.83 \times 10^5 \frac{HP^{5/6}}{N_{drive}^{1.25}}$$

For drive (IDT = 4 and N < 20,000 rpm):

$$Wt_{drive} = K(DPR)^{1/2}$$

where DRP = 139.094 HP/N

$$K = 11.37 \text{ if } DPRL \leq DPR \leq DPRU$$

$$K = 24.1158 \text{ if } DPR > DPRU$$

$$K = 12.0615 \text{ if } DPR < DPRL$$

$$DPRL = e^{-(1.519 + 3.784 \times 10^{-4} N)}$$

$$DPRU = e^{(2.706 - 3.453 \times 10^{-4} N)}$$

For drive (IDT = 4 and N > 20,000 rpm):

$$Wt_{drive} = 2.0113 (HP)^{1/2}$$

For drive (IDT = 5):

$$Wt_{drive} = Wt_{comp}$$

2. Cost Units:

For compressor:

$$CU = 5.5 Wt_{comp}$$

For drive (IDT = 2):

$$CU = 4.0 + 0.6 Wt_{drive}$$

For drive (IDT = 3):

$$CU = 3.0 + Wt_{drive}$$

For drive (IDT = 4):

$$CU = 17.0 + 4.57 Wt_{drive}$$

For drive (IDT = 5):

$$CU = 0.0$$

3. Reliability Index (for compressor and drive):

$$RI = 0.06793$$

4. Development Risk Factor (for compressor and drive):

If $T_{out} \leq 1460^{\circ}R$, $U \leq 1500$ ft/sec, and $N \leq 60,000$ rpm:

$$DR = 1.0$$

If $T_{out} > 1460^{\circ}R$, $U > 1500$ ft/sec, and $N > 60,000$ rpm:

$$DR = [1 + (\frac{T_{out} - 1460}{1000})^2][1 + (\frac{U - 1500}{707})^2]$$

$$[1 + (\frac{N - 60,000}{42,500})^2]$$

5. Weight Standard Error:

For compressor:

$$Wt_d = 0.164 Wt_{comp}$$

For drive (IDT = 2):

$$Wt_d = 0.111 Wt_{drive}$$

For drive (IDT = 3):

$$Wt_d = 0.259 Wt_{drive}$$

For drive (IDT = 4 & 5):

$$Wt_d = 0.2 Wt_{drive}$$

6. Installation Weight Factor (for compressor):

$$Wt_i = 0.205 Wt_{comp}$$

7. Volume:

For compressor:

$$V_{\text{comp}} = 0.6666667 D^4$$

For drive (IDT = 2 and 3):

$$V_{\text{drive}} = 10 W_{\text{dr}}$$

For drive (IDT = 4):

$$V_{\text{drive}} = 20 W_{\text{drive}}$$

8. Efficiency:

For compressor: $\frac{\gamma-1}{\gamma}$

$$\eta_{\text{ad}} = T_{\text{in}} \left[\left(\frac{P_{\text{out}}}{P_{\text{in}}} \right)^{\frac{\gamma}{\gamma-1}} - 1 \right] / (\bar{T}_{\text{out}} - T_{\text{in}})$$

For drive (IDT = 2):

$$\eta = K_1 (1 - 0.281/HP^{0.169})$$

where

$$K_1 = 1.02 \quad (11,000 \text{ rpm} \leq N \leq 22,000 \text{ rpm})$$

$$K_1 = 1.0 \quad (7,200 \text{ rpm} \leq N < 11,000 \text{ rpm})$$

$$K_1 = 0.98 \quad (N < 7,200 \text{ rpm})$$

For drive (IDT = 3):

$$\eta = K_2 (1 - 0.281/HP^{0.169})$$

where $K_2 = f(N)$, and is found by first order interpolation with the following data points:

K_2	N(rpm)
1.04	22,000
1.02	11,500
1.0	7,500
0.98	5,500

For drive (IDT = 4):

$$\eta = 0.82$$

9. Horsepower:

For compressor:

$$HP = 0.02356 W_c p (\bar{T}_{\text{out}} - T_{\text{in}})$$

For drives:

$$HP_{\text{in}} = HP/\eta$$

10. Wheel Diameter (if not specified):

$$D = 12 D_s Q^{1/2} / H^{1/4}$$

where

$$D_s = f(N_s, n_{ad}) \quad (N_s - D_s - n_{ad} \text{ table})$$

$$Q = W/\rho_{in}$$

$$H = 778 C_p T_{in} [(P_{out}/P_{in})^{\frac{Y-1}{Y}} - 1]$$

$$N_s = N Q^{1/2} / H^{3/4}$$

11. Shaft Speed (if not specified):

$$N = N_s H^{3/4} / Q^{1/2}$$

where N_s is found iteratively from a $N_s - D_s - n_{ad}$ table by the generalized Newton-Raphson method.

12. Tip Speed:

$$U = 0.004363 D N$$

Required Tables:

$(N_s - D_s - n_{ad})$ - Table ID (ND,36) - 3 or 33 dimensional, the first independent variable is N_s (argument type 0; argument relative number 59), the second independent variable is n_{ad} (argument type 0; argument relative number 60), and the dependent variable is D_s .

Restrictions:

1. This component can only be used for fluid type 2.

Notes:

1. If both shaft speed and wheel diameter are not specified, the built in limiting shaft speed is used for wheel diameter calculation. If this results in an excessive tip speed (a warning message "tip speed exceeds limiting value" is printed), the built in limiting tip speed is used to determine the wheel diameter and shaft speed iteratively by the generalized Newton-Raphson method.

2. The shaft speed of an AC electric motor drive is defined in the following manner:

motor speed (N _{drive})	compressor speed (N)
5,500	N < 7,200
7,500	7,200 ≤ N < 11,000
11,500	11,000 ≤ N < 22,000

3. If the compressor shaft speed exceeds 22,000 rpm for IDT = 2 & 3, an error message "shaft speed too high for motor drive" is printed. The drive sizing is then bypassed.
4. If the compressor speed exceeds 20,000 rpm for IDT = 4, the drive is sized based on a 20,000 rpm geared unit. A message "hydraulic motor geared" is then printed.
5. The drive is not sized for IDT = 0 & 1.
6. The component printout consists of compressor weight, cost units, reliability index, development risk factor, and the following items:

W - flow rate
PI - inlet pressure
PØ - outlet pressure.
TI - inlet temperature
TØ - outlet temperature
HI - inlet humidity
HØ - outlet humidity
FT - fluid type
FN - fluid properties table relative number
D - wheel diameter
VØL - volume
N - shaft speed
EFF - efficiency
HP - horsepower
U - tip speed
NS - specific speed
DS - specific diameter

If a drive is sized, additional printout consists of drive type and the following items:

EFF - drive efficiency
HP - drive input horsepower
WT - drive weight
CU - drive cost unit
VOL - drive volume (if IDT ≠ 5)

7. If iteration is employed and the iteration dump option is selected, the following information is printed for each iteration:
 - (a) State Variable (S.V.) - specific speed
 - (b) Error Variable (E.V.) - If the conditions described in Note 1 occur, the printed value is the difference of the calculated tip speed and the limiting tip speed. If the wheel diameter is specified, the printed value is the difference of the calculated and the known specific diameter.
8. The convergence limit is 0.001. The upper and the lower limits for the state variable are 10,000 and 1.0, respectively.
9. The values of the wheel diameter and the compressor volume are stored in the general arguments 71 and 72, respectively. If rotational speed is determined, it will be stored in general argument (80 + NST).
10. The built in limiting rotational speed and the limiting tip speed are 60,000 rpm and 1500 ft/sec, respectively. Users may override this by specifying a different value with a COMMON card.

Sizing Component C^{OMP}

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-5	CN	C ^{OMP} component name
8	CC	change code
12	NC	card number
16	NL	leg number
20	NSI	inlet station number
24	NSØ	outlet station number
28	NST	shaft number
32	[WTM]	weight multiplier (optional)
36	[CUM]	cost unit multiplier (optional)
40	[RI]	reliability index (optional)
44	[DRM]	development risk factor multiplier (optional)
48	[D]	wheel diameter = 0 computed > 0 user specified
52	ND	N _s - D _s - n _{ad} table relative number (type 36)
56	ICT	compressor type = 1 high pressure ratio = 2 low pressure ratio
60	-	
64	IDT	drive type = 0 drive not to be sized = 1 turbine = 2 electric motor, AC = 3 electric motor, DC = 4 hydraulic motor = 5 shaft, engine or APU

Sizing Component COND

Condenser

Purpose: This component will compute core dimensions, weight, cost units, reliability index, development risk factor, and core volume for vapor cycle condensers.

Equations:

1. Weight:

(a) Core Weight:

The same equations for heat exchanger core weight are applicable to the condenser core weight calculation.

(See HX subroutine. Note that variables with subscript H (hot side) refer to refrigerant side, and those with subscript C (cold side) refer to heat sink side.)

(b) Condenser Weight:

$$Wt = 3.23126 Wt_{core} / (V)^{0.118}$$

2. Cost Units:

$$CU = 29.7 + 2.72 Wt$$

3. Reliability Index:

$$RI = 0.00306$$

4. Development Risk Factor:

If the average pressure is less than or equal to 314.7 psi:

$$DR = 1.0$$

If the average pressure is greater than 314.7 psi:

$$DR = 1.0 + [(P_{ave} - 314.7)/283.0]^2$$

5. Weight Standard Error:

$$Wt_d = 0.12 Wt$$

6. Installation Weight Factor:

$$Wt_i = 0.205 Wt$$

7. Core Dimensions (if not specified):

Core dimensions L_R , L_S , and L_N are obtained iteratively by the generalized Newton Raphson method from the following three equations:

$$L_R = 6.0 + 0.08 L_N + (0.96 L_N + 0.0064 L_N^2)^{1/2}$$

$$\Delta P_S = \frac{Re_S^2 \mu_S^2 (1 + \psi_S^2)}{1029.568 r_{hs}^2 \rho_{1S}} \left[\frac{f_S L_S N_{PS} (\rho_{1S}/\rho_{2S} + 1)}{2 r_{hs} (1 + \psi_S^2)} + \frac{\rho_{1S}}{\rho_{2S}} - 1 \right]$$

$$\epsilon = 1 - e^{-NTU}$$

where

$$NTU = \frac{L_R L_S L_N}{(WC_p)_S [0.8 (\frac{r_h}{C_p \mu Re j \psi n_o})^{2/3} S + (\frac{r_h}{h \psi}) R]}$$

$$h_R = 0.023148 \text{ Btu/min in.}^{20}\text{F}$$

$$\psi_S = \frac{b_S \beta_S r_{hs}}{b_S + b_R + 2t_{SP}}$$

$$\psi_R = \frac{b_R \beta_R r_{hR}}{b_S + b_R + 2t_{SP}}$$

$$Re_S = \frac{0.8 r_{hs} W_S N_{PS}}{\mu_S L_R L_N \psi_S}$$

$$n_{oS} = 1 - (A_F/A)_S \left[1 - \frac{\tanh(m_S \ell_{eS})}{m_S \ell_{eS}} \right]$$

$$m_S = \left[\frac{1800 C_{pS} \mu_S Re_S j_S (1 + t_{FS}/\xi_S)}{r_{hs} t_{FS}^{2/3} Pr_S h_S} \right]^{1/2}$$

8. Core Volume:

$$V = L_R L_S L_N$$

9. Volume of heat sink liquid inside the core:

$$V_{LS} = \psi_S V$$

Required Tables:

- Table ID (IFS, 31 and IFGR,31) - fin geometry (heat sink side and refrigerant side), refer to HX component for table description.
- f - Table ID (IFS,32) - fin friction factor (heat sink side), refer to HX component for table description.
- j - Table ID (IFS,33) - fin Colburn modulus (heat sink side), refer to HX component for table description.
- t_{SP} - Table ID (ITSP,34) - separation plate thickness, refer to HX component for table description.
- Table ID (IMS,20) - heat sink side fin material properties.
- Table ID (IMR,20) - refrigerant side fin material properties.
- Table ID (IMSP,20) - separation plate material properties.

Restrictions:

1. This component is used for sizing plate fin cross flow vapor cycle condensers.
2. This component can only be used for fluid types 1 and 2 on the heat sink side.
3. The core dimensions (L_R , L_S , L_N) must either be all specified by the user or be all unspecified.

Notes:

1. Two consecutive data cards are required, C_{OND}1 and C_{OND}2. C_{OND}1 card must precede C_{OND}2 card.
2. The same table relative number is used for the fin geometry table, the fin friction factor table, and the fin Colburn modulus table of the heat sink side.
3. Heat sink side fluid properties are evaluated at average temperature, $T_{ave} = (T_{in} + T_{out})/2$, and average pressure, $P_{ave} = (P_{in} + P_{out})/2$.
4. The component printout consists of condenser weight, cost units, reliability index, development risk factor, and the following items:

W - flow rate
PI - inlet pressure
PO - outlet pressure

heat sink side

TI - inlet temperature	}	heat sink side
T \emptyset - outlet temperature		
HI - inlet humidity		
H \emptyset - outlet humidity		
FT - fluid type		
FN - fluid properties table relative number		
LS - heat sink side flow length	}	heat sink side
LR - refrigerant side flow length		
LN - no flow length		
VOL - core volume		
WTC - core weight		
EFF - effectiveness (not printed if core dimensions are specified)	}	heat sink side
NTU - number of heat transfer units (not printed if core dimensions are specified)		
VL - volume of heat sink liquid inside the core (not printed if heat sink is a gas)		

5. Four performance map options are available. They are
 - Option 0 - no maps
 - Option 1 - print maps
 - Option 2 - punch maps
 - Option 3 - print and punch maps
 The maps produced will consist of effectiveness versus heat sink flow rate, and $\sigma\Delta P$ (if a gas) or ΔP (if a liquid) of heat sink side versus heat sink flow rate.
6. If core dimensions are not specified by the user and the iteration dump option is selected, the following information is printed for each iteration:
 - (a) State Variables (S.V.) - the values printed are L_S and L_N , respectively.
 - (b) Error Variables (E.V.) - the two values printed are the difference of the calculated and the given heat sink side pressure drops, and the difference of the calculated and the given effectivenesses, respectively.

- (c) The four values printed after the error variables are Re_S , NTU, ϵ , and η_{OS} , respectively.
7. The convergence limits for pressure drop and effectiveness are 0.01 and 0.0001, respectively. The upper and the lower limits for the state variables are 100.0 and 0.1, respectively.
 8. The dry air rated heat sink outlet temperature is used for effectiveness calculation.
 9. Values of L_S , L_R , L_N , V , V_{LS} , and W_t are stored in general arguments 71, 72, 73, 74, 75, and 76, respectively.

Sizing Component CØND

Data Card Format

1. CØND1 Data Card

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-5	CN	CØND1 component name
8	CC	change code
12	NC	card number
16	NLS	leg number, heat sink side
20	NSIS	inlet station number, heat sink side
24	NSOS	outlet station number, heat sink side
28	NSØR	outlet station number, refrigerant side
32	[WTM]	weight multiplier (optional)
36	[CUM]	cost unit multiplier (optional)
40	[RI]	reliability index (optional)
44	[DRM]	development risk factor multiplier (optional)
48	MAP	performance map options = 0 no maps = 1 print maps = 2 punch maps = 3 print and punch maps

2. C_{OND}2 Data Card

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-5	CN	C _{OND} 2 component name
8	CC	change code
12	NC	card number
16	[LS]	core flow length, heat sink side = 0 computed > 0 user specified
20	[LR]	core flow length, refrigerant side = 0 computed > 0 user specified
24	[LN]	no flow length = 0 computed > 0 user specified
28	NPS	number of passes, heat sink side
32	IMS	fin material properties table relative number, heat sink side (type 20)
36	IMR	fin material properties table relative number, refrigerant side (type 20)
40	IFS	fin variable table relative number, heat sink side (types 31, 32, 33)
44	IFGR	fin geometry table relative number, refrigerant side (type 31)
48	IMSP	separation plate material properties table relative number (type 20)
52	ITSP	separation plate thickness table relative number (type 34)

Sizing Component DSEP

Dust Separator

Purpose: This component will compute weight, cost units, reliability index, development risk factor, and volume for dust separators. The available dust separator types are:

IDT Type

1 static

2 self-cleaning, high pressure

Equations:

1. Weight:

For IDT = 1:

$$\therefore W_t = 0.195 W^{0.79}$$

For IDT = 2:

$$W_t = 0.5 + 0.0346 W$$

2. Cost Units:

For IDT = 1:

$$CU = 5.4 + 0.71 W_t$$

For IDT = 2:

$$CU = 9.11 + 1.07 W_t$$

3. Reliability Index:

$$RI = 0.04135$$

4. Development Risk Factor:

$$DR = 1.0$$

5. Weight Standard Error:

For IDT = 1:

$$W_{t_d} = 0.04 W_t$$

For IDT = 2:

$$W_{t_d} = 0.1225 W_t$$

6. Installation Weight Factor:

$$W_{t_i} = 0.205 W_t$$

7. Volume:

For IDT = 1:

$$V = 7.3 W_t^{1.28}$$

For IDT = 2:

$$V = 19.0 W_t^{1.5}$$

Restrictions:

1. This component can only be used for fluid type 2.

Notes:

1. The component printout consist of weight, cost units, reliability index, development risk factor, and the following items:

W - flow rate

PI - inlet pressure

P \emptyset - outlet pressure

TI - inlet temperature

T \emptyset - outlet temperature

HI - inlet humidity

H \emptyset - outlet humidity

FT - fluid type

FN - fluid properties table relative number

VOL - volume

2. The value of volume is stored in general argument 71.

Sizing Component DSEP

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-4	CN	DSEP component name
8	CC	change code
12	NC	card number
16	NL	leg number
20	NSI	inlet station number
24	NSO	outlet station number
28	[WTM]	weight multiplier (optional)
32	[CUM]	cost unit multiplier (optional)
36	[RI]	reliability index (optional)
40	[DRM]	development risk factor multiplier (optional)
44	IDT	dust separator type = 1 static = 2 self-cleaning, high pressure

Sizing Component EJECT

Ejector

Purpose: This component will compute weight, cost units, reliability index, development risk factor, effective nozzle area, and volume for an ejector assembly, which includes a mixing tube, high pressure tubing, and nozzles.

Equations:

1. Weights:

$$Wt = 20.0 (2.5 B + 2) t \rho / 1728$$

2. Cost Units:

$$CU = 17.96 + 5.36 Wt^{0.64} + 0.139 Wt^{1.65}$$

3. Reliability Index:

$$RI = 0.00177$$

4. Development Risk Factor:

$$DR = 1.0$$

5. Weight Standard Error:

$$Wt_d = 0.25 Wt$$

6. Installation Weight Factor:

$$Wt_i = 0.205 Wt$$

7. Volume:

$$V = 38.6 B$$

8. Effective Nozzle Area:

If flow is choked:

$$A = (W/28.3725) [\gamma P_{in} \rho_{in} (\frac{2}{1+\gamma})^{\frac{\gamma+1}{\gamma-1}}]^{1/2}$$

If flow is not choked:

$$A = (W/40.1248) (\frac{\gamma}{\gamma-1} P_{in} \rho_{in})^{1/2} [(\frac{P_{out}}{P_{in}})^{\frac{2}{\gamma}} - (\frac{P_{out}}{P_{in}})^{\frac{\gamma+1}{\gamma}}]^{1/2}$$

9. Number of Single - Nozzle Ejector Modules:

<u>A</u>	0.0 ~ 3.0	3.0 ~ 5.0	5.0 ~ 7.0	7.0 ~ 9.0	etc
B	2	4	6	8	etc

Required Tables:

- Table ID (IMT,20) - mixing tube material properties.

Restrictions:

1. This component can only be used for fluid type 2.

Notes:

1. The component printout consists of weight, cost units, reliability index, development risk factor, and the following items:

W - primary flow rate

PI - primary flow inlet pressure

PØ - primary flow outlet pressure

TI - primary flow inlet temperature

TØ - primary flow outlet temperature

HI - primary flow inlet humidity

HØ - primary flow outlet humidity

FT - primary flow fluid type

FN - primary flow fluid properties table relative number

AN - effective nozzle area

VØL - volume

2. The value of volume is stored in general argument 71.

Sizing Component EJECT

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-5	CN	EJECT component name
8	CC	change code
12	NC	card number
16	NL	leg number, primary flow
20	NSI	inlet station number, primary flow
24	NSØ	outlet station number, primary flow
28	[WTM]	weight multiplier (optional)
32	[CUM]	cost unit multiplier (optional)
36	[RI]	reliability index (optional)
40	[DRM]	development risk factor multiplier (optional)
44	[THK]	mixing tube wall thickness
48	IMT	material properties table relative number (type 20)

Sizing Component EVAP

Evaporator

Purpose: This component will compute core dimensions, weight, cost units, reliability index, development risk factor, and core volume for vapor cycle evaporators.

Equations:

1. Weight:

(a) Core Weight:

The same equations for heat exchanger core weight are applicable to the evaporator core weight calculation. (See HX component.) Note that variables with subscript C (cold side) refer to refrigerant side, and those with subscript H (hot side) refer to heat source side.

(b) Evaporator Weight:

$$Wt = 3.23126 Wt_{core} / (V)^{0.118}$$

2. Cost Units:

$$CU = 29.7 + 2.72 Wt$$

3. Reliability Index:

$$RI = 0.00216$$

4. Development Risk Factor:

If the average pressure is less than or equal to 314.7 psi:

$$DR = 1.2$$

If the average pressure is greater than 314.7 psi:

$$DR = 1.2 [1.0 + \left(\frac{P_{ave} - 314.7}{283} \right)^2]$$

5. Weight Standard Error:

$$Wt_d = 0.12 Wt$$

6. Installation Weight Factor:

$$Wt_i = 0.205 Wt$$

7. Core Dimensions (if not specified):

Core dimensions L_R , L_S and L_N are obtained iteratively by the generalized Newton Raphson method from the following three equations:

$$L_R = 3.0 + 0.18 L_N + (1.08 L_N + 0.0324 L_N^2)^{1/2}$$

$$\Delta P_S = \frac{Re_S^2 \mu_S^2 (1 + \psi_S^2)}{1029.568 r_{hs}^2 \rho_{1S}} \left[\frac{f_S L_S N_{PS} (\rho_{1S}/\rho_{2S} + 1)}{2 r_{hs} (1 + \psi_S^2)} + \frac{\rho_{1S}}{\rho_{2S}} - 1 \right]$$

$$\epsilon = 1 - e^{-NTU}$$

where

$$NTU = \frac{L_R L_S L_N}{(WC_p)_S [0.8 \left(\frac{r_h^2 Pr^{2/3}}{C_p \mu \cdot Re \cdot j \cdot \psi \cdot n_o} \right)_S + \left(\frac{r_h}{h \psi} \right)_R]}$$

$$h_R = 0.017361 \text{ Btu/min in}^2 {}^\circ\text{F} = \cancel{150.} \text{ BTU/hr ft}^2 {}^\circ\text{F}$$

$$\psi_S = \frac{b_S \beta_S r_{hs}}{b_S + b_R + 2t_{SP}}$$

$$\psi_R = \frac{b_R \beta_R r_{hR}}{b_S + b_R + 2t_{SP}}$$

$$Re_S = \frac{0.8 r_{hs} W_S N_{PS}}{\mu_S L_R L_N \psi_S}$$

$$n_{OS} = 1 - (A_F/A)_S \left[1 - \frac{\tanh(m_S \ell_{eS})}{m_S \ell_{eS}} \right]$$

$$m_S = \left[\frac{1800 C_{ps} \mu_S Re_S j_S (1 + t_{FS}/\xi_S)}{r_{hs} t_{FS} Pr_S^{2/3} k_S} \right]^{1/2}$$

8. Core Volume:

$$V = L_R L_S L_N$$

9. Volume of heat source liquid inside the core:

$$V_{LS} = \psi_S V$$

Required Tables:

- Table ID (IFS,31 and IFGR,31) - fin geometry (heat source side and refrigerant side), refer to HX component for table description.
- f - Table ID (IFS,32) - fin friction factor (heat source side), refer to HX component for table description.
- j - Table ID (IFS,33) - fin Colburn modulus (heat source side), refer to HX component for table description.
- t_{SP} - Table ID (ITSP,34) - separation plate thickness, refer to HX component for table description.
- Table ID (IMS,20) - heat source side fin material properties.
- Table ID (IMR,20) - refrigerant side fin material properties.
- Table ID (IMSP,20) - separation plate material properties.

Restrictions:

1. This component is used for sizing plate fin cross flow vapor cycle evaporators.
2. This component can only be used for fluid types 1 and 2 on the heat source side.
3. The core dimensions (L_R , L_S , L_N) must either be all specified by the user or be all unspecified.

Notes:

1. Two consecutive data cards are required, EVAP1 and EVAP2. The EVAP1 card must precede the EVAP2 card.
2. The same table relative number is used for the fin geometry table, the fin friction factor table, and the fin Colburn modulus table of the heat source side.
3. Heat source side fluid properties are evaluated at average temperature, $T_{ave} = (T_{in} + T_{out})/2$, and average pressure, $P_{ave} = (P_{in} + P_{out})/2$.
4. The component printout consists of evaporator weight, cost units, reliability index, development risk factor, and the following items:

W	- flow rate	}	heat source side
PI	- inlet pressure		
P \emptyset	- outlet pressure		
TI	- inlet temperature		
T \emptyset	- outlet temperature		
HI	- inlet humidity		
H \emptyset	- outlet humidity		
FT	- fluid type		
FN	- fluid properties table relative number	}	heat source side
LS	- heat source side flow length		
LR	- refrigerant side flow length		
LN	- no flow length		
V \emptyset L	- core volume		
WTC	- core weight		
EFF	- effectiveness (not printed if core dimensions are specified)		
NTU	- number of heat transfer units (not printed if core dimensions are specified)		
VL	- volume of heat source liquid inside the core (not printed if heat source is a gas)		

5. Four performance map options are available. They are

- Option 0 - no maps
- Option 1 - print maps
- Option 2 - punch maps
- Option 3 - print and punch maps

The maps produced will consist of effectiveness versus heat source flow rate, and $\sigma\Delta P$ (if a gas) or ΔP (if a liquid) of heat source side versus heat source flow rate.

6. If core dimensions are not specified by the user and the iteration dump option is selected, the following information is printed for each iteration:
- (a) State Variables (S.V.) - the two values printed are L_S and L_N , respectively.

- (b) Error Variables (E.V.) - the two values printed are the difference of the calculated and the given heat source side pressure drops, and the difference of the calculated and the given effectivenesses, respectively.
- (c) The four values printed after the error variables are Re_S , NTU, ϵ , and η_{OS} , respectively.
7. The convergence limits for pressure drop and effectiveness are 0.01 and 0.0001, respectively. The upper and the lower limits for state variables are 100.0 and 0.1, respectively.
8. The dry air rated heat source outlet temperature is used for effectiveness calculation.
9. Values of L_S , L_R , L_N , V , V_{LS} , and W_t are stored in general arguments 71, 72, 73, 74, 75, and 76, respectively.

Sizing Component EVAP

Data Card Format

1. EVAP1 Data Card

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-5	CN	EVAP1 component name
8	CC	change code
12	NC	card number
16	NLS	leg number, heat source side
20	NSIS	inlet station number, heat source side
24	NSØS	outlet station number, heat source side
28	NSØR	outlet station number, refrigerant side
32	[WTM]	weight multiplier (optional)
36	[CUM]	cost unit multiplier (optional)
40	[RI]	reliability index (optional)
44	[DRM]	development risk factor multiplier (optional)
48	MAP	performance map options = 0 no maps = 1 print maps = 2 punch maps = 3 print and punch maps

2. EVAP2 Data Card

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-5	CN	EVAP2 component name
8	CC	change code
12	NC	card number
16	[LS]	core flow length, heat source side = 0 computed > 0 user specified
20	[LR]	core flow length, refrigerant side = 0 computed > 0 user specified
24	[LN]	no flow length = 0 computed > 0 user specified
28	NPS	number of passes, heat source side
32	IMS	fin material properties table relative number, heat source side (type 20)
36	IMR	fin material properties table relative number, refrigerant side (type 20)
40	IFS	fin variable table relative number, heat source side (types 31, 32, 33)
44	IFGR	fin geometry table relative number, refrigerant side (type 31)
48	IMSP	separation plate material properties table rela- tive number (type 20)
52	ITSP	separation plate thickness table relative number (type 34)

Sizing Component FAN

Fan

Purpose: This component will compute weight, cost units, reliability index, blade tip diameter, and volume for a fan (including a drive). The available types of drives are:

IDT	Type
2	electric motor, AC
3	electric motor, DC
4	hydraulic motor
6	tip turbine

Equations:

1. Weight:

For IDT = 2 & 3:

$$Wt = 0.26 D^2$$

For IDT = 4:

$$Wt = 0.178 D^2$$

For IDT = 6:

$$Wt = 0.109 D_E^2$$

where D_E = effective blade tip diameter

2. Cost Units:

For IDT = 2:

$$CU = 14.29 + 0.46 Wt$$

For IDT = 3:

$$CU = 4.87 + 0.76 Wt$$

For IDT = 4:

$$CU = 58.587 + 1.725 Wt$$

For IDT = 6:

$$CU = 5.5 Wt$$

3. Reliability Index:

For IDT = 2 & 3:

$$RI = 0.00546$$

For IDT = 4:

$$RI = 0.01162$$

For IDT = 6:

$$RI = 0.00707$$

4. Development Risk Factor:

$$DR = 1.0$$

5. Weight Standard Error:

For IDT = 2 & 3:

$$Wt_d = 0.25 Wt$$

For IDT = 4:

$$Wt_d = 0.109 Wt$$

For IDT = 6:

$$Wt_d = 0.199 Wt$$

6. Installation Weight Factor:

$$Wt_i = 0.205 Wt$$

7. Volume:

For IDT = 2, 3 & 4:

$$V = 1.3 D^3$$

For IDT = 6:

$$V = 0.8666667 D_E^3$$

8. Blade Tip Diameter (if not specified):

$$D = 16.8 \left(\frac{W}{\rho N} \right)^{1/3} \quad (1)$$

9. Effective Blade Tip Diameter (for IDT = 6):

$$D_E = D + 2 h \quad (2)$$

The effective blade tip diameter is found interatively by the generalized Newton Raphson method from Equation (1), Equation (2), turbine $N_s - D_s - n_{ad}$ table, and turbine $N_s - h/D - n_{ad}$ table.

10. Fan Horsepower:

$$HP = 0.02356 W C_p (T_{out} - T_{in})$$

11. Fan Adiabatic Efficiency:

$$n_{ad} = \frac{T_{in} \left[\left(\frac{P_{out}}{P_{in}} \right)^{\frac{1}{\gamma}} - 1 \right]}{T_{out} - T_{in}}$$

Required Tables (for IDT = 6 only):

$(N_s - D_s - n_{ad})$ - Table ID (ITS,37) - 3 or 33 dimensional, the first independent variable is N_s (argument type 0; argument relative number 59), the second independent variable is n_{ad} (argument type 0; argument relative number 60), and the dependent variable is D_s .

$(N_s - h/D - n_{ad})$ - Table ID (ITS,38) - 3 or 33 dimensional, the first independent variable is N_s (argument type 0; argument relative number 59), the second independent variable is n_{ad} (argument type 0, argument relative number 60), and the dependent variable is h/D .

Restrictions:

1. This component can only be used for fluid type 2.
2. Shaft speed must be specified for IDT = 2, 3, & 4. For IDT = 6, the effective blade tip diameter and shaft speed must be either both specified or both unspecified.

Notes:

1. The component printout consists of weight, cost units, reliability index, development risk factor, and the following items:
 - W - flow rate
 - PI - inlet pressure
 - P \emptyset - outlet pressure
 - TI - inlet temperature
 - T \emptyset - outlet temperature
 - HI - inlet humidity
 - H \emptyset - outlet humidity
 - FT - fluid type
 - FN - fluid properties table relative number
 - D - blade tip diameter (if IDT = 2, 3, & 4) or effective blade tip diameter (if IDT = 6)
 - V \emptyset L - volume
 - N - shaft speed
 - EFF - fan adiabatic efficiency
 - HP - fan fluid horsepower

Additional printout consists of drive type and the following items:

EFF - drive efficiency
HP - drive input horsepower
W - turbine flow rate
PI - turbine inlet pressure
P \emptyset - turbine outlet pressure
TI - turbine inlet temperature
T \emptyset - turbine outlet temperature
HI - turbine inlet humidity
H \emptyset - turbine outlet humidity
FT - fluid type
FN - fluid properties table relative number

} for IDT = 6 only

2. If iteration is employed and the iteration dump option is selected, the following information is printed for each iteration:
 - (a) State Variable (S.V.) - specific speed.
 - (b) Error Variable (E.V.) - the difference of the calculated turbine wheel diameter and the calculated effective fan blade tip diameter.
3. The convergence limit is 0.001. The upper and lower limits for the state variable are 1000.0 and 1.0 respectively.
4. The values of the blade tip diameter (or effective blade tip diameter) and the volume are stored in general argument 71 and 72, respectively.
5. For IDT = 6, the limiting rotational speed and the limiting tip speed are 60,000 rpm and 1,500 ft/sec, respectively. Users may override this by specifying a different value with a COMMON card.

Sizing Component FAN

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-3	CN	FAN component name
8	CC	change code
12	NC	card number
16	NLF	fan leg number
20	NSIF	fan inlet station number
24	NSOF	fan outlet station number
28	NST	shaft number
32	[WTM]	weight multiplier (optional)
36	[CUM]	cost unit multiplier (optional)
40	[RI]	reliability index
44	[DRM]	development risk factor multiplier (optional)
48	[D]	fan tip diameter (if IDT = 2, 3, & 4) or turbine tip diameter (if IDT = 6) = 0 computed > 0 user specified
52	IDT	drive type = 2 electric motor, AC = 3 electric motor, DC = 4 hydraulic motor = 6 tip turbine
56	NLT	turbine leg number (not required if IDT ≠ 6)
60	NSIT	turbine inlet station number (not required if IDT ≠ 6)
64	NSOT	turbine outlet station number (not required if IDT ≠ 6)
68	ITS	turbine sizing characteristic table relative number (types 37 & 38, not required if IDT ≠ 6)

Sizing Component HEATER

Resistance Heater

Purpose: This component will compute weight, cost units, reliability index, development risk factor, volume, and flow length for resistance heaters.

Equations:

1. Weight:

$$Wt = \rho V / 1728$$

where

$$\rho = 20.0$$

$$(V \leq 794.3 \text{ in}^3)$$

$$\rho = 374.11/V^{0.4384}$$

$$(V > 794.3 \text{ in}^3)$$

2. Cost Units:

$$CU = 2.08 Wt$$

3. Reliability Index:

$$RI = 0.01109$$

4. Development Risk Factor:

$$DR = 1.0$$

5. Weight Standard Error:

$$Wt_d = 0.12 Wt$$

6. Installation Weight Factor:

$$Wt_i = 0.205 Wt$$

7. Volume:

$$V = 1.21444 W C_p (\bar{T}_{out} - T_{in})$$

8. Flow Length (if not specified and the performance map is required: The flow length L is obtained iteratively by the generalized Newton Raphson method from the following equation:

$$\Delta P = \frac{Re^2 \mu^2 (1 + \psi^2)}{1029.568 r_h^2 \rho_1} \left[\frac{L_f (\rho_1 / \rho_2 + 1)}{2r_h (1 + \psi^2)} + \frac{\rho_1}{\rho_2} - 1 \right]$$

where

$$Re = 0.8 r_h W L / (\mu V \psi)$$

$$\psi = \frac{b \beta r_h}{b + b' + 0.012}$$

b' = heater element height

Required Tables (if the map option is not 0):

- Table ID (IF,31) - fin geometry, refer to HX component for table description.
- f - Table ID (IF,32) - fin friction factor, refer to HX component for table description.

Restriction:

1. This component can only be used for fluid type 2.

Notes:

1. The same table relative number is used for the fin geometry table and the fin friction factor table.
2. Fluid properties are evaluated at average temperature,
 $T_{ave} = (T_{in} + T_{out})/2$, and average pressure, $P_{ave} = (P_{in} + P_{out})/2$.
3. The component printout consists of weight, cost units, reliability index, development risk factor, and the following items:

W - flow rate

PI - inlet pressure

P \emptyset - outlet pressure

TI - inlet temperature

T \emptyset - outlet temperature

HI - inlet humidity

H \emptyset - outlet humidity

FT - fluid type

FN - fluid properties table relative number

VOL - volume

L - flow length (if the map option is not 0)

4. If iteration is employed and the iteration dump option is selected, the following information is printed for each iteration:
 - (a) State Variable (S.V.) - flow length
 - (b) Error Variable (E.V.) - the difference of the calculated and the given pressure drops.
 - (c) RE - Reynolds number
5. The convergence limit is 0.01. The upper and the lower limits of the state variables are 100.0 and 0.1 respectively.

6. Four performance map options are available. They are:

Option 0 - no map

Option 1 - print map

Option 2 - punch map

Option 3 - print and punch map.

The maps produced will give values of $\sigma\Delta P$ versus flow rate.

7. The values of the flow length and the volume are stored in general arguments 71 and 72, respectively.

Sizing Component HEATER

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-6	CN	HEATER component name
8	CC	change code
12	NC	card number
16	NL	leg number
20	NSI	inlet station number
24	NSO	outlet station number
28	[WTM]	weight multiplier (optional)
32	[CUM]	cost unit multiplier (optional)
36	[RI]	reliability index (optional)
40	[DRM]	development risk factor multiplier (optional)
44	[L]	flow length (not required if MAP = 0) = 0 computed > 0 user specified
48	[IHEL]	heater element height (not required if MAP = 0)
52	IF	fin variable tables relative number (types 31 and 32, not required if MAP = 0)
56	MAP	performance map options = 0 no map = 1 print map = 2 punch map = 3 print and punch map

SECTION XX

B11 - REPLACEMENT

SIZING COMPONENT HX

Heat Exchanger

Purpose: This component will compute core dimensions, weight, cost units, reliability index, development risk factor, and core volume for heat exchangers.

Equations:

1. Weight:

(a) Core Weight:

If $\rho_{SP} < 460$:

$$Wt_{core} = \frac{\rho_{FH} V_H + \rho_{FC} V_C + \rho_{SP} t_{SP} L_{CH} (N_H N_{SH} + N_C N_{SC} + 1.0)}{1728}$$

If $\rho_{SP} \geq 460$:

$$Wt_{core} = \frac{\rho_{FH} V_H + \rho_{FC} V_C + \rho_{SP} t_{SP} L_{CH} (N_H N_{SH} + N_C N_{SC} + 1.0)}{1728}$$

$$+ 6.44 \times 10^{-4} L_{CH} (N_H N_{SH} + N_C N_{SC} + 1.0)$$

$$N_H = \frac{L_N - b_C - 2t_{SP}}{b_H + b_C + 2t_{SP}}$$

$$N_C = N_H + 1$$

$$V_H = N_H [b_H - (N_{SH} - 1) t_{SP}] L_C L_H$$

$$V_C = N_C [b_C - (N_{SC} - 1) t_{SP}] L_C L_H$$

$$\rho_F = \frac{t_F^{pp} \text{metal} [B + 1/p - (9 - 2\pi)t_F]}{B} \quad - \text{rectangular fin}$$

$$\rho_F = \frac{t_F^{pp} \text{metal} [B + (\pi - 2)/2p - t_F]}{B} \quad - \text{rounded fin}$$

$$\rho_F = \frac{t_F^{pp} \text{metal} [S + 3t_F \phi]}{B} \quad - \text{triangular fin}$$

$$B = \frac{b - (N_S - 1) t_{SP}}{N_S}$$

$$\theta = 2 \cos^{-1} \left(\frac{3t_F}{a + 3t_F} \right)$$

$$a = \frac{-1 + 6p^2 t_F B + (1 - 12 p^2 t_F^2 + p^2 B^2)^{1/2}}{1/(3t_F) - 12p^2 t_F}$$

$$S = 1/(p \cos(\theta/2)) - 6t_F p (B + 2a)$$

(b) Heat Exchanger Weight:

$$W_t = 3.23126 W_{\text{core}} / (V)^{0.118}$$

2. Cost Units:

$$CU = 2.08 W_t$$

3. Reliability Index:

$$RI = 0.00256 \text{ (gas/gas HX)}$$

$$RI = 0.00328 \text{ (gas/liquid HX)}$$

$$RI = 0.00106 \text{ (liquid/liquid HX)}$$

4. Development Risk Factor:

For gas/gas HX:

$$DR = 1.0 \quad (T \leq 1760^{\circ}\text{R}, P \leq 314.7 \text{ psi})$$

$$DR = 1.0 + \left(\frac{T - 1760}{425} \right)^2 + \left(\frac{P - 314.7}{283} \right)^2 \quad (T > 1760^{\circ}\text{R}, P > 314.7 \text{ psi})$$

For gas/liquid or liquid/liquid HX:

$$DR = 1.0 \quad (P \leq 314.7 \text{ psi})$$

$$DR = 1.0 + \left(\frac{P - 314.7}{283} \right)^2 \quad (P > 314.7 \text{ psi})$$

5. Weight Standard Error:

$$W_{\text{d}} = 0.12 W_t$$

6. Installation Weight Factor:

$$W_i = 0.205 W_t$$

7. Core Dimensions (if not specified):

Core dimensions L_H , L_C and L_N are obtained iteratively by the generalized Newton Raphson method from the following three equations:

$$\Delta P_H = \frac{Re_H^2 \mu_H^2 (1 + \psi_H^2)}{1029.568 r_{hH}^2 \rho_{1H}} \left[\frac{f_H L_H N_{PH} (\rho_{1H}/\rho_{2H} + 1)}{2r_{hH} (1 + \psi_H^2)} + \frac{\rho_{1H}}{\rho_{2H}} - 1 \right]$$

$$\Delta P_C = \frac{Re_C^2 \mu_C^2 (1 + \psi_C^2)}{1029.568 r_{hC}^2 \rho_{1C}} \left[\frac{f_C L_C N_{PC} (\rho_{1C}/\rho_{2C} + 1)}{2r_{hC} (1 + \psi_C^2)} + \frac{\rho_{1C}}{\rho_{2C}} - 1 \right]$$

$\epsilon = f$ (NTU, CC) (heat transfer effectiveness table)

where

$$NTU = r_{hH} W_H L_H N_{PH}' / [(WC'_p)_{min} \psi_H \mu_H Re_H]$$

$$\left(\frac{r_{hH}^2 \Pr_H^{2/3}}{n_{OH} C'_p H \mu_H \psi_H j_H Re_H} + \frac{r_{hC}^2 \Pr_C^{2/3}}{n_{OC} C'_p C \mu_C \psi_C j_C Re_C} \right)$$

$$CC = (WC'_p)_{min} / (WC'_p)_{max} \quad (\text{both fluids mixed or both fluids unmixed})$$

$$CC = (WC'_p)_{mixed} / (WC'_p)_{unmixed} \quad (\text{one fluid mixed})$$

$$\psi_H = \frac{b_H \beta_H r_{hH} N_H}{(b_H + b_C + 2t_{SP}) N_H + b_C + 2t_{SP}}$$

$$\psi_C = \frac{b_C \beta_C r_{hC} N_C}{(b_H + b_C + 2t_{SP}) N_H + b_C + 2t_{SP}}$$

$$Re_H = \frac{0.8 r_{hH} W_H N_{PH}}{\mu_H L_C L_N \psi_H}$$

$$Re_C = \frac{0.8 r_{hC} W_C N_{PC}}{\mu_C L_H L_N \psi_C}$$

$$n_{OH} = 1 - \left(\frac{A_F}{A} \right)_H \left[1 - \frac{\tanh(m_H \ell_{eH})}{m_H \rho_{eH}} \right]$$

$$\eta_{OC} = 1 - \left(\frac{A_F}{A} \right)_C [1 - \frac{\tanh(m_C \ell_{eC})}{m_C \ell_{eC}}]$$

$$m_H = \left[\frac{1800 C'_{pH} \mu_H \text{Re}_H j_H (1 + t_{FH}/\xi_H)}{r_{hH} t_{FH} \text{Pr}_H^{2/3} k_H} \right]^{1/2}$$

$$m_C = \left[\frac{1800 C'_{pC} \mu_C \text{Re}_C j_C (1 + t_{FC}/\xi_C)}{r_{hC} t_{FC} \text{Pr}_C^{2/3} k_C} \right]^{1/2}$$

$$\text{SHR}_H = (T_{H_{in}} - \bar{T}_{H_{out}}) / (T_{H_{in}} - T_{H_{out}})$$

$$\text{SHR}_C = (T_{C_{in}} - \bar{T}_{C_{out}}) / (T_{C_{in}} - T_{C_{out}})$$

$$C'_{pH} = C_{pH} \cdot \text{SHR}$$

$$C'_{pC} = C_{pC} \cdot \text{SHR}$$

8. Core Volume:

$$V = L_H L_C L_N$$

9. Volume of liquid inside the core:

$$V_{LH} = \psi_H V$$

$$V_{LC} = \psi_C V$$

Required Tables:

- Table ID (IF1, 31 and IF2, 31) - fin geometry (side 1 and side 2), -1 dimensional, 9 table values are in the following order:

A_F/A - fin area/total area

p - fin pitch

b - stack height

r_h - hydraulic radius

t_F - fin thickness

- β - total heat transfer area/volume of one side
 ξ - effective length in flow direction
 FS - fin shape code
 = 1.0 rectangular fin
 = 2.0 rounded fin
 = 3.0 triangular fin
 N_S - number of sandwiches
 = 1.0 single
 = 2.0 double
 = 3.0 triple
 etc.
 f - Table ID (IF1, 32 and IF2, 32) - fin friction factor
 (side 1 and side 2), 2 dimensional, the independent variable is Reynolds number (argument type 0; argument relative number 21), and the dependent variable is friction factor.
 j - Table ID (IF1, 33 and IF2, 33) - fin Colburn modulus
 (side 1 and side 2), 2 dimensional, the independent variable is Reynolds number (argument type 0; argument relative number 21), and the dependent variable is Colburn modulus.
 ϵ - Table ID (IEF, 5) - heat transfer effectiveness, 3 or 33 dimensional, the first independent variable is NTU (argument type 0; argument relative number 55), the second independent variable is $(WC_p)_{min}/(WC_p)_{max}$ or $(WC_p)_{mixed}/(WC_p)_{unmixed}$ (argument type 0; argument relative number 56), and the dependent variable is heat transfer effectiveness.
 t_{SP} - Table ID (ITSP, 34) - separation plate thickness.
 - Table ID (IM1, 20) - side 1 fin material properties.
 - Table ID (IM2, 20) - side 2 fin material properties
 - Table ID (IMSP, 20) - separation plate material properties.

Restrictions:

1. This component is used for sizing a plate fin cross flow heat exchanger.
2. The component can only be used for fluid types 1 and 2.
3. The core dimensions (L_H , L_C , L_N) must either be all specified by the user or be all unspecified.

Notes:

1. Three consecutive data cards are required, HX, HX1 and HX2. The HX card must precede HX1 and HX2 cards.
2. For each side of the heat exchanger, the same table relative number is used for the fin geometry table, the fin friction factor table and the fin Colburn modulus table.
3. Fluid properties are evaluated at average temperature,
 $T_{ave} = (T_{in} + T_{out})/2$, and average pressure,
 $P_{ave} = (P_{in} + P_{out})/2$.
4. The component printout consists of heat exchanger weight, cost units, reliability index, development risk factor, and the following items:

W - flow rate
PI - inlet pressure
PO - outlet pressure
TI - inlet temperature
TO - outlet temperature
HI - inlet humidity
HO - outlet humidity
FT - fluid type
FN - fluid properties table
relative number
LH - hot side flow length
LC - cold side flow length
LN - no flow length
VOL - core volume
WTC - core weight
EFF - heat transfer effectiveness (not printed if core dimensions are specified)

hot side and
cold side

NTU - number of heat transfer units (not printed if core dimensions are specified)

VLH - volume of hot side liquid inside the core (not printed if hot side fluid is a gas)

VLC - volume of cold side liquid inside the core (not printed if cold side fluid is a gas)

5. Four performance map options are available. They are:

Option 0 - no maps

Option 1 - print maps

Option 2 - punch maps

Option 3 - print and punch maps

The maps produced will consist of the following:

- side 1 temperature effectiveness versus flow rates for side 1 and side 2.
- side 1 and side 2 $\sigma\Delta P$ (for gas) or ΔP (for liquid) versus flow rate.

6. If core dimensions are not specified by the user and the iteration dump option is selected, the following information is printed for each iteration:

- State Variables (S.V.) - the three values printed are L_H , L_C , and L_N , respectively.
- Error Variables (E.V.) - the three values printed are the difference of the calculated and the given hot side pressure drops, the difference of the calculated and the given cold side pressure drops, and the difference of the calculated and the given heat transfer effectiveness, respectively.
- The six values printed after the error variables are Re_H , Re_C , NTU, ϵ , η_{OH} , and η_{OC} , respectively.

7. The convergence limits for pressure drops and heat transfer effectiveness are 0.001 and 0.0001, respectively. The upper and lower limits for state variables are 100.0 and 0.1, respectively.
8. Values of L_H , L_C , L_N , V , V_{LH} , and V_{LC} are stored in general arguments 71, 72, 73, 74, 75 and 76, respectively.

Sizing Component HX

Data Cards Format

1. HX Data Card

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-2	CN	HX component name
8	CC	change code
12	NC	card number
16	[WTM]	weight multiplier (optional)
20	[CUM]	cost unit multiplier (optional)
24	[RI]	reliability index (optional)
28	[DRM]	development risk factor multiplier (optional)
32	[LN]	no flow length = 0 computed > 0 user specified
36	IMSP	separation plate material table relative number (type 20)
40	ITSP	separation plate thickness table <u>relative number (type 34)</u>
44	IEF	effectiveness table relative number (type 5) ϵ vs. NTU and $\frac{(WC_p)_{min}}{(WC_p)_{max}}$
48	MAP	performance map options = 0 no maps = 1 print maps = 2 punch maps = 3 print and punch maps
52	IFLT	flow type = 1 both sides unmixed = 2 both sides mixed = 3 side 1 mixed, side 2 unmixed = 4 side 1 unmixed, side 2 mixed

1. Al (.012)
2. Steel/Nickel (.026)

1. 6061 Aluminum
2. AISI 347 Steel
3. Brazed
4. Magnesium Nickel
5. Necone 600

of passes
and # 2 three
crossflow
three 6 passes

rel
55 56
p 23

20x10x10

2. HX1 Data Card

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-3	CN	HX1 component name
8	CC	change code
12	NC	card number
16	NL1	leg number, side 1
20	NSI1	inlet station number, side 1
24	NSØ1	outlet station number, side 1
28	[L1]	core flow length, side 1 = 0 computed > 0 user specified <u>number of passes, side 1</u>
32	NP1	
36	IM1	<u>fin material properties table relative number, side 1 (type 20)</u>
40	IF1	<u>fin variable tables relative number, side 1 (types 31, 32, 33)</u>

3. HX2 Data Card

1-3	CN	HX2 component name
8	CC	change code
12	NC	card number
16	NL2	leg number, side 2
20	NSI2	inlet station number, side 2
24	NSØ2	outlet station number, side 2
28	[L2]	core flow length, side 2 = 0 computed > 0 user specified <u>number of passes, side 2</u>
32	NP2	
36	IM2	<u>fin material properties table relative number, side 2 (type 20)</u>
40	IF2	<u>fin variable tables relative number, side 2 (types 31, 32, 33)</u>

B12

Sizing Component INIT

Initialization

Purpose: This component is used to initialize a known component value.

The available types of values are:

ITYP	Value
1	weight
2	cost units
3	reliability index
4	development risk factor
5	weight standard error
6	electrical power
7	shaft power
8	hydraulic power
9	bleed air extraction
10	fuel flow rate (lb/hr)
11	drag
12	expendable coolant weight

Equations:

For ITYP = 1, 2, 3, and 6 through 12

$$(\quad)_{\text{system}} = (\quad)_{\text{system}} + \text{Value}$$

For ITYP = 4

$$\text{DR}_{\text{system}} = \text{DR}_{\text{system}} \times \text{Value}$$

For ITYP = 5

$$(Wt_d)_{\text{system}}^2 = (Wt_d)_{\text{system}}^2 + (\text{Value})^2$$

Notes:

1. This component may be used to define an additional system development risk factor.
2. This component must be used to define bleed air extraction if system penalty calculation is required.
3. The component printout consists of initialization type and value.

Sizing Component INIT

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-4	CN	INIT component name
8	CC	change code
12	NC	card number
16	ITYP	type of initialized value = 1 weight = 2 cost unit = 3 reliability index = 4 development risk factor = 5 weight standard error = 6 electrical power = 7 shaft power = 8 hydraulic power = 9 bleed air extraction = 10 fuel flow rate = 11 drag = 12 expendable coolant weight
20	[VALUE]	initialized value

Sizing Component INSLTN

Insulation

Purpose: This component will compute weight, cost units, reliability index, development risk factor, and insulation thickness for ducting and compartment insulation. The available insulation types and insulation methods are:

INT Insulation Type

- 1 bleed air line
- 2 conditioned air distribution line
- 3 compartment

INM Insulation Method

- 1 wrap around (for INT = 1 & 2)
blanket (for INT = 3)
- 2 integral blanket (for INT = 1 & 2)
blanket with air gap (for INT = 3)
- 3 removable blanket with radiation reflector (for INT = 1)
- 4 air gap radiation shield (for INT = 1)

Equations:

1. Weight:

For INT = 1 and INM = 1 & 3:

$$\begin{aligned} Wt = L & [(1.9667 \times 10^{-3} \rho_i D + 7.0 \times 10^{-4} \rho_i \\ & + 7.7 \times 10^{-2})(t - 0.25) + 4.25 \times 10^{-4} \rho_i D \\ & + 0.01 D + 2.0833 \times 10^{-4} \rho_i + 1.59167 \times 10^{-2})] \quad (1) \end{aligned}$$

For INT = 1 and INM = 2:

$$\begin{aligned} Wt = L & [(5.5 \times 10^{-3} t + 5.8833 \times 10^{-3}) D \\ & - 0.015 t - 5.8333 \times 10^{-4}] \quad (2) \end{aligned}$$

For INT = 1 and INM = 4:

$$Wt = L[(6.0667 \times 10^{-3} D + 8.6 \times 10^{-3}) (t - 0.25) \\ + 2.35 \times 10^{-3} D + 1.0667 \times 10^{-3})] \quad (3)$$

For INT = 2 and INM = 1:

$$Wt = L(D + t)(6.1078 \times 10^{-4} + 1.0917 \times 10^{-3} t) \quad (4)$$

For INT = 2 and INM = 2:

$$Wt = L[(D + t)(9.725 \times 10^{-3} + 1.275 \times 10^{-3} t) \\ - 4.8625 \times 10^{-3} D] \quad (5)$$

For INT = 3:

$$Wt = A [3.646 \times 10^{-4} t / CF + 5.104 \times 10^{-4}] \quad (6)$$

2. Cost Units:

For INT = 1:

$$CU = L[0.009583t(D + t) + 0.04425]$$

For INT = 2:

$$CU = 0.002378 DL$$

For INT = 3:

$$CU = A(6.944 \times 10^{-4} + 7.9167 \times 10^{-4} t)$$

3. Reliability Index:

$$RI = 0.00001$$

4. Development Risk Factor:

$$DR = 1.0$$

5. Weight Standard Error:

For INT = 1 and INM = 1 & 3:

$$Wt_d = 0.055 Wt$$

For INT = 1 and INM = 2:

$$Wt_d = 0.149 Wt$$

For INT = 1 and INM = 4:

$$Wt_d = 0.167 Wt$$

For INT = 2:

$$Wt_d = 0.15 Wt$$

For INT = 3:

$$Wt_d = 0.15 Wt$$

6. Installation Weight Factor:

$$Wt_i = 0.205 Wt$$

7. Thickness (if not specified):

For INT = 1 and INM = 1 & 2:

$$t_{\text{calc}} = 0.045 (T_D - 460) / (T_S - 0.2 T_D - 416)$$

For INT = 1 and INM = 3:

$$t_{\text{calc}} = 0.064 (T_D - 460) / (T_S - 0.2 T_D - 416)$$

For INT = 1 and INM = 4:

$$t_{\text{calc}} = 0.06 (T_D - 460) / (T_S - 0.2 T_D - 416)$$

For INT = 1 and all INM

$$t = t_{\text{calc}} \text{ if } 0 < t_{\text{calc}} \leq 1$$

$$t = 1 \text{ if } t_{\text{calc}} \leq 0 \text{ or } t_{\text{calc}} > 1$$

For INT = 2:

$$t = \frac{D}{2} (e^c - 1)$$

where

$$c = 2\pi k_i \left(\frac{T_{in} + T_{out}}{2} - T_A \right) / (q/L)$$

$$\frac{q}{L} = 60 W C_p (T_{in} - T_{out})/L$$

For INT = 3:

$$t = 12 (LF) k_i (T_S - T_C) / (q_T/A)$$

where

LF = heat leak factor

$$= 2.625$$

$$q_T/A = q_c/A + q_r/A$$

$$= 1.5 \left(\frac{P_A}{14.7} \right)^{1/2} (T_C - T_A) + 0.9 (1.714 \times 10^{-9}) (T_C^4 - T_A^4)$$

Required Tables:

- Table ID(IMT,20) - insulation material properties.

Restrictions:

1. Equation (1) applies only to line diameters greater than 1.5 inches.
2. Equation (2) applies only to 3pcf insulation with a 0.006 inch thick stainless steel outer cover and a 0.002 inch thick aluminum radiation reflector.
3. Equation (3) applies only to line diameters approximately one inch or greater.
4. Equation (4) applies only to 0.6 pcf insulation with 0.125 lb per square yard cover sheets.
5. Equation (5) applies only to 0.6 pcf insulation with 5 ounces per square yard cover sheets.

Notes:

1. The insulation weights for flange clamps and expansion bellows are included in the installation weight factor.
2. The component printout consists of weight, cost units, reliability index, development risk factor, and the following item:
THK - insulation thickness or air gap size
3. The insulation material properties table input is always required although it is not always required in the computation. For INT = 1 and INM = 4, the user may input a dummy material properties table.
4. The insulation conductivity, k_i , is evaluated at an average temperature of insulation inner and outer surfaces. The insulation density, ρ_i , is constant. C_p is evaluated at the inlet temperature, T_{in} .

Sizing Component INSLTN

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-6	CN	INSLTN component name
8	CC	change code
12	NC	card number
16	[WTM]	weight multiplier (optional)
20	[CUM]	cost unit multiplier (optional)
24	[RI]	reliability index (optional)
28	[DRM]	development risk factor multiplier (optional)
32	INT	insulation type = 1 bleed air line = 2 distribution line = 3 compartment
36	INM	insulation method = 1 wrap around (for INT = 1 & 2) blanket (for INT = 3) = 2 integral blanket (for INT = 1 & 2) blanket with air gap (for INT = 3) = 3 removable blanket with radiation reflector (for INT = 1) = 4 air gap radiation shield (for INT = 1)

If INT = 1

40	[THK]	insulation thickness or air gap size = 0 computed > 0 user specified
44	[D]	line outer diameter
48	[L]	line length
52	[TD]	line temperature (not required if [THK] > 0) = 0 use fluid inlet temperature > 0 user specified
56	NSI	inlet station number (not required if [THK] > 0 or [TD] > 0)
60	[TS]	insulation outer surface temperature (not required if [THK] > 0)
64	-	
68	IMT	insulation material properties table relative number (type 20)

If INT = 2

40	[THK]	insulation thickness = 0 computed > 0 user specified
44	[D]	line outer diameter
48	[L]	line length
52	[TA]	ambient temperature (not required if [THK] > 0)
56	NL	leg number
60	NSI	inlet station number
64	NSO	outlet station number
68	IMT	insulation material properties table relative number (type 20)

If INT = 3

40	[THK]	insulation thickness = 0 computed > 0 user specified
44	[A]	insulation area
48	[PC]	compartment pressure (not required if [THK] > 0)
52	[TC]	compartment temperature (not required if [THK] > 0)
56	[TW]	compartment wall temperature (not required if [THK] > 0)
60	[TS]	insulation outer surface temperature (not required if [THK] > 0)
64	[CF]	compression factor
68	IMT	insulation material properties table relative number (type 20)

B14

Sizing Component LINE

Line

Purpose: This component will compute weight, cost units, reliability index, development risk factor, and inside diameter for lines. The available types of lines are:

ILT Type

- 1 high pressure bleed line, aluminum
- 2 high pressure bleed line, steel
- 3 low pressure distribution or ram air line
- 4 refrigeration package line
- 5 liquid coolant line, aluminum

Equations:

1. Weight:

For ILT = 1:

$$Wt = 0.34 D t L$$

For ILT = 2:

$$Wt = (2.4 D + 1.25) t L$$

For ILT = 3:

$$Wt = 0.0137 D L \quad (D \leq 3.5)$$

$$Wt = (0.0179 D - 0.0147) L \quad (D > 3.5)$$

For ILT = 4:

$$Wt = 0.7 (2.4 D + 1.25) t L$$

For ILT = 5:

$$Wt = 0.3071 (D + t) t L$$

2. Cost Units:

For ILT = 1 & 5:

$$CU = (0.0097 D + 0.0191) L$$

For ILT = 2 & 4:

$$CU = (0.103 D + 0.026) L \quad (D \leq 2.5)$$

$$CU = (0.231 D - 0.295) L \quad (D > 2.5)$$

For ILT = 3:

$$CU = (0.011 D + 0.0295) L \quad (D \leq 3.5)$$

$$CU = (0.0254 D - 0.0233) L \quad (D > 3.5)$$

3. Reliability Index:

For ILT = 1, 2 & 4:

$$RI = 0.01164$$

For ILT = 3:

$$RI = 0.00291$$

For ILT = 5:

$$RI = 0.00285$$

4. Development Risk Factor:

$$DR = 1.0$$

5. Weight Standard Error:

$$Wt_d = 0.15 Wt + 0.158 Wt_i$$

6. Installation Weight:

For ILT = 1, 2, & 3:

$$Wt_i = 3.45 \times 10^{-4} (D L)^{1.58}$$

For ILT = 4 & 5:

$$Wt_i = 0.0$$

7. Liquid Volume (for ILT = 5 only):

$$V = \frac{\pi}{4} D^2 L$$

8. Inside Diameter (if not specified):

$$\Delta P = 1.008 \times 10^{-3} \left(\frac{4 f L}{D} + C K_t \right) \frac{W^2}{\rho D^4} - \frac{\rho Z}{1728}$$

where

Z = inlet elevation - outlet elevation

The above equation is solved for inside diameter iteratively by the generalized Newton Raphson method.

9. Wall Thickness (if not specified):

For ILT = 1:

$$t = 0.035 \quad (D \leq 3.5)$$

$$t = 0.042 \quad (D > 3.5)$$

For ILT = 2 & 4:

$$t = 0.00677 D + 0.00322$$

For ILT = 5:

$$t = 0.01 \quad (D \leq 1.0)$$

$$t = 0.028 \quad (D > 1.0)$$

For ILT = 3:

Wall thickness is not required.

Required Tables (if inside diameter is not specified):

f - Table ID(IF,3) - friction factor

K_t - Table ID (IKT,2) - bend pressure loss factor

Restrictions:

1. This component can only be used for fluid types 1 (if ILT = 1) or 2 (if ILT ≠ 1)

Notes:

1. The component printout consists of weight, cost units, reliability index, development risk factor, and the following items:

W - flow rate

PI - inlet pressure

P \emptyset - outlet pressure

TI - inlet temperature

T \emptyset - outlet temperature

HI - inlet humidity

H \emptyset - outlet humidity

FT - fluid type

FN - fluid properties table relative number

D - inside diameter

THK - wall thickness (for ILT ≠ 3 only)

V - velocity

M - Mach number (for ILT ≠ 5 only)

VL - liquid volume (for ILT = 5 only)

2. If a line inside diameter is not specified and the iteration dump option is selected, the following information is printed for each iteration:

(a) State Variable (S.V.) - line inside diameter.

(b) Error Variable (E.V.) - the difference of the calculated and the given pressure drops.

(c) RE - Reynolds number.

3. The convergence limit is 0.01. The upper and lower limits for the state variable are 10.0 and 0.1, respectively.
4. The values of inside diameter, velocity, Mach number, and liquid volume are stored in general arguments 71, 72, 73, and 74, respectively.
5. Reynolds number is available in general argument 21, and diameter is available in general argument 57, for the f and K_t table lookup.
6. If diameter is not specified, the f or the K_t term can be optional, but not both.

Sizing Component LINE

<u>Card Column</u>	<u>Symbol</u>	<u>Data Card Format</u>
1-4	CN	LINE component name
8	CC	change code
12	NC	card number
16	NL	leg number
20	NSI	inlet station number
24	NSØ	outlet station number
28	[WTM]	weight multiplier (optional)
32	[CUM]	cost unit multiplier (optional)
36	[RI]	reliability index (optional)
40	[DRM]	development risk factor multiplier (optional)
44	[D]	inside diameter = 0 computed > 0 user specified
48	[L]	line length
52	[THK]	wall thickness (not required if ILT = 3) = 0 computed > 0 user specified
56	[Z]	difference in elevation between line inlet and outlet
60	IF	friction factor table relative number (type 3, optional)
64	[C]	correction factor for loss of bends other than 90° (optional)
68	IKT	90° bend pressure loss factor table relative number (type 2, optional)
72	ILT	line type = 1 high pressure bleed line, aluminum = 2 high pressure bleed line, steel = 3 low pressure distribution or ram air line = 4 refrigeration package line = 5 liquid coolant line, aluminum

Sizing Component MISC

Miscellaneous Operations

Purpose: This component will compute any value desired using basic mathematical expressions. The result of the operation may be a general argument or a parameter table value. Variables one and two may be a general argument, a flow rate, a pressure, a temperature, a humidity, a parameter table value or a table lookup value.

Options: The operations which are a function of one variable are: equality, absolute value, square root, base ten logarithm, natural logarithm, ten to a power, and e to a power. The operations which are a function of two variables are: addition, subtraction, multiplication, division, the minimum of two variables, the maximum of two variables, variable one to the variable two power, average value, and a code which indicates that variable one is either less than, equal to, or greater than variable two.

Equations:

```
if IOPR ≤ 0
    VX = f (V1)
if IOPR > 0
    VX = f (V1, V2)
```

Notes:

1. The component printout consists of the following items:

VX - result variable value

IVXT - result variable type

IVXN - result variable relative number

IOPR - operation code

V1 - variable value

IV1T - variable type

IV1N - variable relative number

V2 - variable value

IV2T - variable type

IV2N - variable relative number

} variable one

} variable two
(printed if IOPR > 0)

Sizing Component MISC

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-4	CN	MISC component name
8	CC	change code
12	NC	card number
16	IVXT	result type
		= 0 - general argument
		= -1 - parameter value
20	NVX	result relative number If IVXT
		= 0 - general argument number
		= -1 - parameter index
24	IVLT	variable one type
		= 0 - general argument
		= 1 - flow rate
		= 2 - pressure
		= 3 - temperature
		= 4 - humidity
		= -1 - parameter value
		= -2 - table (type 99)
28	NVL	variable one relative number if IVLT
		= 0 - general argument relative number
		= 1 - leg number
		= 2 -
		= 3 - } station number
		= 4 -
		= -1 - parameter index
		= -2 - table relative number (type 99)
32	IOPR	operation if IOPR > 0 VX = f(V1, V2)
		= 1 - V1 + V2
		= 2 - V1 - V2
		= 3 - V1 x V2
		= 4 - V1 ÷ V2
		= 5 - minimum value (V1, V2)
		= 6 - maximum value (V1, V2)
		= 7 - V1V2
		= 8 - (V1 + V2)/2

(B15.2)(MISC)

```

= 9 - 0 if V1 = V2
    1 if V1 > V2
   -1 if V1 < V2
if IOPR = 0
    VX = V1
if IOPR < 0 VX = f(V1)
    = -1 - |V1|
    = -2 - √V1
    = -3 - log V1
    = -4 - ln V1
    = -5 - 10V1
    = -6 - eV1

```

36

IV2T

variable two type
 (same as IV1T)

40

NV2

variable two relative number
 (same as NV1)

Sizing Component PUMP

Pump

Purpose: This component will compute weight, cost units, reliability index, development risk factor, and volume for a pump package.

It will also compute the efficiency and the horsepower for a pump and its drive. The available pump types are:

- | | |
|-----|--------------|
| IPT | Type |
| 1 | vane or gear |
| 2 | centrifugal |

Equations:

1. Weight:

For pump and motor:

For IPT = 1:

$$Wt_p = 2.1$$

(DPR < 0.088 in³/rev)

$$Wt_p = 10.63 (DPR)^{2/3}$$

(DPR ≥ 0.088 in³/rev)

For IPT = 2:

$$Wt_p = 2.1$$

(DPR < 0.028 in³/rev)

$$Wt_p = 22.92 (DPR)^{2/3}$$

(DPR ≥ 0.028 in³/rev)

where

$$DPR = \frac{1728 W}{(\rho N)}$$

For reservoir:

$$Wt_r = 0.131 V_f^{2/3}$$

where

$$V_f = Wt_{liquid} \left(\frac{1/\rho_{high}}{temp} - \frac{1/\rho_{low}}{temp} \right)$$

$$Wt_{liquid} = \rho_{in} V_{liquid}$$

For filter:

$$Wt_{filter} = 0.55 + 0.9575 W/\rho$$

For pump package:

$$Wt = 1.125 (Wt_p + Wt_r + Wt_{filter}) + Wt_{liquid}$$

2. Cost Units:

$$CU = 81.2 + 1.05 (Wt_p + Wt_r + Wt_{filter})$$

3. Reliability Index:

$$RI = 0.04016$$

4. Development Risk Factor:

$$DR = 1.0$$

5. Weight Standard Error:

$$Wt_d = (0.0361 Wt_p^2 + 0.0551 Wt_r^2 + 0.0441 Wt_{filter}^2)^{1/2}$$

6. Installation Weight Factor:

$$Wt_i = 0.205 (Wt_p + Wt_r + Wt_{filter})$$

7. Volume:

$$V = 33.04 (Wt_p + Wt_r + Wt_{filter})$$

8. Horsepower:

(a) Fluid horsepower:

$$HP_f = 4.3636 \times 10^{-3} (P_{out} - P_{in}) W/\rho_{in}$$

(b) Shaft Horsepower:

$$HP_S = HP_f + WC_P (T_{out} - T_{in})/42.416$$

9. Efficiency:

$$\eta = HP_f / HP_S$$

Restrictions:

1. This component can only be used for fluid type 1.
2. This component is used for sizing an electric motor driven pump only.
3. Shaft speed must be specified by the user.

Notes:

1. Refer to sizing subroutine C0MP for drive type, and equations (DC motor) for drive efficiency and horsepower.
2. The volumetric change of the total amount of liquid in the liquid loop is computed at a high temperature of 710°R and a low temperature of 395°R.
3. The component printout consists of weight (including weight of liquid in the liquid loop subsystem), cost units, reliability index, and development risk factor of the pump package, and the following items:

W - flow rate

PI - inlet pressure

PØ - outlet pressure

TI - inlet temperature

TØ - outlet temperature

HI - inlet humidity

HØ - outlet humidity

FT - fluid type
FN - fluid properties table relative number

WTP - weight of pump and motor

WTR - weight of reservoir

WTL - weight of filter

V_OL - volume of the pump package

N - shaft speed

EFF - pump efficiency

HP - pump shaft horsepower

Additional printout consists of drive type and the following items:

EFF - drive efficiency

HP - drive input horsepower

4. The value of package volume is stored in general argument 71.

Sizing Component PUMP

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-4	CN	PUMP component name
8	CC	change code
12	NC	card number
16	NL	leg number
20	NSI	inlet station number
24	NSO	outlet station number
28	NST	shaft number
32	[WTM]	weight multiplier (optional)
36	[CUM]	cost unit multiplier (optional)
40	[RI]	reliability index (optional)
44	[DRM]	development risk factor multiplier (optional)
48	[VL]	liquid volume in the whole liquid loop
52	IPT	pump type = 1 vane or gear = 2 centrifugal

Sizing Component RIN

Ram Air Inlet

Purpose: This component will compute weight, cost units, reliability index, development risk factor, inlet area, diffuser exit area, volume, and drag for inlets. The available inlet types are:

INLT	Type
1	flush inlet
2	scoop inlet
3	nose inlet
4	engine duct inlet

Equations:

1. Weight:

$$Wt = \rho t_e (8 A_I + S) / 1728$$

where t_e is the effective wall thickness of inlet and diffuser.

2. Cost Units:

$$CU = 13.0 + 0.139 Wt^{1.65}$$

3. Reliability Index:

$$RI = 0.0001$$

4. Development Risk Factor:

$$DR = 1.0$$

5. Weight Standard Error:

$$Wt_d = 0.25 Wt$$

6. Installation Weight Factor:

$$Wt_i = 0.205 Wt$$

7. Inlet Area:

For INLT = 1:

$$A_I = \frac{3.6823 W}{\rho_\infty V_\infty} \quad (\text{for flight condition, } V_\infty > 0)$$

$$A_I = \frac{W \sqrt{T_\infty}}{8.9376 P_\infty} \quad (\text{for sea level static condition, } V_\infty = 0)$$

For INLT = 2:

$$A_I = \frac{3.4286 W}{\rho_\infty V_\infty} \quad (\text{for subsonic flight condition})$$

$$A_I = \frac{3 W}{\rho_\infty V_\infty} \quad (\text{for supersonic flight condition})$$

$$A_I = \frac{W \sqrt{T_\infty}}{8.9376 P_\infty} \quad (\text{for sea level static condition})$$

For INLT = 3:

$$A_I = \frac{3 W}{\rho_\infty V_\infty} \quad (\text{for flight condition})$$

$$A_I = \frac{W \sqrt{T_\infty}}{8.9376 P_\infty} \quad (\text{for sea level static condition})$$

For INLT = 4:

$$A_I = \frac{2.4 W}{0.3 \rho_I c_I}$$

where c_I is the speed of sound at the inlet.

8. Diffuser Exit Area:

$$A_D = \frac{2.4 W}{0.15 \rho_D c_D}$$

where c_D is the speed of sound at the diffuser exit.

9. Diffuser Surface Area:

$$S = \frac{A_D - A_I}{0.10453}$$

10. Volume:

$$V = 1.89 A_I^{1.5} + \frac{(\sqrt{A_D} - \sqrt{A_I})(A_I + A_D + \sqrt{A_I A_D})}{0.5589}$$

11. Drag:

$$D = \frac{W V_\infty}{1932}$$

Required Tables:

- Table ID (IM,20) - material properties

Restrictions:

1. This component can only be used for fluid type 2.

Notes:

1. An inlet consists of an inlet section and a diffuser.
2. The outlet station refers to the station at the end of the mixing section.
3. The component printout consists of weight, cost units, reliability index, development risk factor, and the following items:

W - flow rate

PI - free stream pressure

PØ - outlet pressure.

TI - free stream temperature

TØ - outlet temperature

HI - free stream humidity

HØ - outlet humidity

FT - fluid type

FN - fluid properties table relative number

AI - inlet area

AD - diffuser exit area

VØL - volume

DRAG - inlet drag

4. If $S < 0.0$, the diffuser is not sized. A message is printed, and A_D is set to zero.

5. The free stream fluid, free stream pressure, temperature, Mach number, and velocity must be defined.

6. The values of volume and inlet area are stored in general argument 71 and 72, respectively.

Sizing Component RIN

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-3	CN	RIN component name
8	CC	change code
12	NC	card number
16	NL	leg number
20	NSO	mixing section outlet station number
24	[WTM]	weight multiplier (optional)
28	[CUM]	cost unit multiplier (optional)
32	[RI]	reliability index (optional)
36	[DRM]	development risk factor multiplier (optional)
40	[AI]	inlet area = 0 computed => 0 user specified
44	[THK]	effective wall thickness
48	IM	material properties table relative number (type 20)
52	INLT	inlet type = 1 flush inlet = 2 scoop inlet = 3 nose inlet = 4 engine duct inlet

Sizing Component R_{OUT}

Ram Air Outlet

Purpose: This component will compute weight, cost units, reliability index, development risk factor, outlet area, and volume for an outlet of a ram air duct.

Equations:

1. Weight:

$$Wt = 4.5 \rho t_e (1.0 + 1.69 A_0^{1/2} + 1.2 A_0)$$

2. Cost Units:

$$CU = 13.0 + 0.139 Wt^{1.65}$$

3. Reliability Index:

$$RI = 0.00010$$

4. Development Risk Factor:

$$DR = 1.0$$

5. Weight Standard Error:

$$Wt_d = 0.25 Wt$$

6. Installation Weight Factor:

$$Wt_i = 0.205 Wt$$

7. Volume

$$V = 0.85 A_0^{1.5}$$

8. Outlet Area:

$$A_0 = (W/40.1248C_D) \left(\frac{\gamma}{\gamma-1} P_0 \rho_0 \right)^{1/2} \left[(PR)^\gamma - (PR)^{\frac{\gamma+1}{\gamma-1}} \right]^{1/2}$$

$$PR = P_{\text{ambient}} / P_0$$

Required Table:

- Table ID (IM,20) - material properties

Restriction:

1. This component can only be used for fluid type 2.

Notes:

1. The component printout consists of weight, cost units, reliability index, development risk factor, and the following items:

W - flow rate
 P_I - inlet pressure
 P_O - outlet pressure
 T_I - inlet temperature
 T_O - outlet temperature
 H_I - inlet humidity
 H_O - outlet humidity
 F_T - fluid type
 F_N - fluid properties table relative number
 A_O - outlet area
 $V_O L$ - volume

2. The values of volume and outlet area are stored in general arguments 71 and 72 respectively.

Sizing Component R \emptyset UT

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-4	CN	R \emptyset UT component name
8	CC	change code
12	NC	card number
16	NL	leg number
20	NSI	inlet station number
24	NS \emptyset	outlet station number
28	[WTM]	weight multiplier (optional)
32	[CUM]	cost unit multiplier (optional)
36	[RI]	reliability index (optional)
40	[DRM]	development risk factor multiplier (optional)
44	[A \emptyset]	outlet area = 0 computed > 0 user specified
48	[THK]	effective wall thickness
52	IM	material properties table relative number (table 20)
56	[CD]	discharge coefficient (not required if [A \emptyset] > 0)

Sizing Component TURB

Turbine

Purpose: This component will compute weight, cost units, reliability index, development risk factor, volume, wheel diameter and/or shaft speed for an air cycle turbine (centrifugal or axial).

Equations:

1. Weight:

$$Wt = 0.4 D^2$$

2. Cost Units:

$$CU = 5.5 Wt$$

3. Reliability Index:

$$RI = 0.0$$

4. Development Risk Factor:

$$DR = 1.0$$

5. Weight Standard Error:

$$Wt_d = 0.164 Wt$$

6. Installation Weight Factor:

$$Wt_i = 0.205 Wt$$

7. Volume:

$$V = 0.6666667 D^4$$

8. Efficiency:

$$\eta_{ad} = \frac{(T_{in} - \bar{T}_{out})}{[T_{in} \{1.0 - (P_{out}/P_{in})^{\frac{Y-1}{Y}}\}]}$$

9. Horsepower:

$$HP = 0.02356 Wc_p (T_{in} - \bar{T}_{out})$$

10. Wheel Diameter (if not specified):

$$D = 12 D_s^{1/2} / H^{1/4}$$

where

$$D_s = f(N_s, \eta_{ad}) \quad (N_s - D_s - \eta_{ad} \text{ table})$$

$$Q = W/\rho_{in}$$

$$H = 778 C_p T_{in} [1.0 - (P_{out}/P_{in})^{\frac{1}{\gamma}}]$$

$$N_s = N Q^{1/2} / H^{3/4}$$

11. Shaft speed (if not specified):

$$N = N_s H^{3/4} / Q^{1/2}$$

where N_s is found iteratively from $N_s - D_s - n_{ad}$ table by the generalized Newton Raphson method.

12. Tip speed:

$$U = 0.004363 D N$$

Required Tables:

$(N_s - D_s - n_{ad})$ - Table ID (ND,37) - 3 or 33 dimensional, the first independent variable is N_s (argument type 0; argument relative number 59), the second independent variable is n_{ad} (argument type 0; argument relative number 60), and the dependent variable is D_s .

Restrictions:

1. This component can only be used for fluid type 2.

Notes:

1. If both shaft speed and wheel diameter are not specified, the built in limiting shaft speed is used for wheel diameter calculation. If this results in an excessive tip speed (a warning message "tip speed exceeds limiting value" is printed), the built in limiting tip speed is used to determine the wheel diameter and shaft speed iteratively by the generalized Newton Raphson method.
2. The component printout consists of weight, cost units, reliability index, development risk factor, and the following items:

W - flow rate

PI - inlet pressure

PQ - outlet pressure

TI - inlet temperature
 T_O - outlet temprature
 HI - inlet humidity
 H_O - outlet humidity
 FT - fluid type
 FN - fluid properties table relative number
 D - wheel diameter
 V_OL - volume
 N - shaft speed
 EFF - efficiency
 HP - horsepower
 U - tip speed
 NS - specific speed
 DS - specific diameter

3. The reliability index and the development risk factor are accounted for in the component driven by the turbine.
4. If iteration is employed and the iteration dump option is selected, the following information is printed for each iteration:
 - (a) State Variable (S.V.) - specific speed
 - (b) Error Variable (E.V.) - if the conditions described in Note 1 occur, the printed value is the difference of the calculated tip speed and the limiting tip speed. If the wheel diameter is specified, the printed value is the difference of the calculated and the known specific diameters.
5. The convergence limit is 0.001. The upper and the lower limits for the state variable are 10,000 and 1.0, respectively.
6. The values of the wheel diameter and the compressor volume are stored in the general arguments 71 and 72, respectively. If rotational speed is determined, it will be stored in general argument (80 + NST).
7. The built in limiting rotational speed and the limiting tip speed are 60,000 rpm and 2,000 ft/sec, respectively. Users may override this by specifying a different value with a C_OM_M_ON card.

Sizing Component TURB

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-4	CN	TURB component name
8	CC	change code
12	NC	card number
16	NL	leg number
20	NSI	inlet station number
24	NSØ	outlet station number
28	NST	shaft number
32	[WTM]	weight multiplier (optional)
36	[CUM]	cost unit multiplier (optional)
40	[RI]	reliability index (optional)
44	[DRM]	development risk factor multiplier (optional)
48	[D]	wheel diameter = 0 computed > 0 user specified
52	ND	$N_s - D_s - n_{ad}$ table relative number (type 37)
56	ITT	turbine type = 1 centrifugal = 2 axial

Sizing Component VALVE
Valve

Purpose: This component will compute weight, cost units, reliability index, development risk factor, and inside diameter for valves.

IVT	Type
1	pneumatic butterfly
2	electric butterfly
3	45° stem poppet
4	60° stem poppet
5	90° stem poppet
6	split flapper check

Equations:

1. Weight:

For IVT = 1 through 6 and fluid type 1: liquid

$$Wt = 1.1$$

For IVT = 1 and fluid type 2: gas

$$Wt = 2.68 D - 1.0 \quad (\text{steel})$$

$$Wt = 0.7 D + 0.35 \quad (\text{aluminum}, D \leq 2.5)$$

$$Wt = 1.26 D - 1.05 \quad (\text{aluminum}, D > 2.5) \quad 2.10 \text{ lbs} @ 2.5" D$$

For IVT = 2 and fluid type 2:

$$Wt = 1.14 D + 0.9 \quad (\text{steel})$$

$$Wt = 0.56 D + 1.1 \quad (\text{aluminum})$$

For IVT = 3, 4, 5 and fluid type 2:

$$Wt = 2.04 D + 0.225 \quad (\text{steel})$$

$$Wt = 0.833 D - 0.05 \quad (\text{aluminum})$$

For IVT = 6 and fluid type 2:

$$Wt = 0.625 D - 0.625 \quad (\text{steel})$$

$$Wt = 0.33 D - 0.38 \quad (\text{aluminum})$$

2. Cost Units:

For IVT = 1 through 6 and fluid type 1:

$$CU = 5.4$$

For IVT = 1, 2 and fluid type 2:

$$CU = 1.13 (D Wt)^{1.11} \quad (D Wt > 17.5)$$

$$CU = 27.0 \quad (D Wt \leq 17.5)$$

For IVT = 3, 4, 5 and fluid type 2:

$$CU = 8.4 (D Wt)^{0.192} \quad (D Wt > 0.1)$$

$$CU = 5.4 \quad (D Wt \leq 0.1)$$

For IVT = 6 and fluid type 2:

$$CU = 3.7 (D Wt)^{0.193} \quad (D Wt > 0.1)$$

$$CU = 2.37 \quad (D Wt \leq 0.1)$$

3. Reliability Index:

For IVT = 1 through 6 and fluid type 1:

$$RI = 0.00219$$

For IVT = 1 and fluid type 2:

$$RI = 0.0461$$

For IVT = 2 and fluid type 2:

$$RI = 0.07649$$

For IVT = 3, 4, 5 and fluid type 2:

$$RI = 0.05291$$

For IVT = 6 and fluid type 2:

$$RI = 0.0571$$

4. Development Risk Factor:

$$DR = 1.0 \quad (T \leq 1460^{\circ}R, P \leq 364.7 \text{ psi})$$

$$DR = 1.0 + \left(\frac{T-1460}{450} \right)^2 + \left(\frac{P-364.7}{585} \right)^2$$

$$(T > 1460^{\circ}R, P > 364.7 \text{ psi})$$

5. Weight Standard Error:

For IVT = 1 through 6 and fluid type 1:

$$Wt_d = 0.303 Wt \quad (\text{steel})$$

$$Wt_d = 0.348 Wt \quad (\text{aluminum})$$

For IVT = 1 and fluid type 2:

$$Wt_d = 0.35 Wt \quad (\text{steel})$$

$$Wt_d = 0.339 Wt \quad (\text{aluminum})$$

For IVT = 2 and fluid type 2:

$$Wt_d = 0.201 Wt \quad (\text{steel})$$

$$Wt_d = 0.174 Wt \quad (\text{aluminum})$$

For IVT = 3, 4, 5 and fluid type 2:

$$Wt_d = 0.303 Wt \quad (\text{steel})$$

$$Wt_d = 0.348 Wt \quad (\text{aluminum})$$

For IVT = 6 and fluid type 2:

$$Wt_d = 0.3 Wt \quad (\text{steel})$$

$$Wt_d = 0.359 Wt \quad (\text{aluminum})$$

6. Installation Weight Factor:

$$Wt_i = 0.0$$

7. Diameter (if not specified):

$$D = [1.008 \times 10^{-3} K_t W^2 / (\rho \Delta P)]^{0.25}$$

where $K_t = 0.43$ for IVT = 1 & 2

= 2.74 for IVT = 3

= 3.4 for IVT = 4

= 6.5 for IVT = 5

= 0.68 for IVT = 6

Restrictions:

1. This component can only be used for fluid types 1 and 2.

Notes:

1. The component printout consists of weight, cost units, reliability index, development risk factor and the following items:

W - flow rate

PI - inlet pressure

P \emptyset - outlet pressure

TI - inlet temperature

T \emptyset - outlet temperature

HI - inlet humidity

H \emptyset - outlet humidity

FT - fluid type

FN - fluid properties table relative number

D - inside diameter

KT - pressure loss coefficient

V - velocity

M - Mach number (for fluid type 2 only)

2. Steel valve equations are used if inlet temperature is greater than or equal to T_{LM} . Aluminum valve equations are used if it is less than T_{LM} . The built in T_{LM} is 960°R. Users may override this by specifying a different T_{LM} with a COMMON card.

3. The minimum weight for IVT = 1 is 0.5. For IVT = 3 through 5, the minimum weight is 0.3. For IVT = 6, it is 0.25.
4. The values of inside diameter, velocity, and Mach number are stored in the general arguments 71, 72, and 73, respectively.

Sizing Component VALVE

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-5	CN	VALVE component name
8	CC	change code
12	NC	card number
16	NL	leg number
20	NSI	inlet station number
24	NSØ	outlet station number
28	[WTM]	weight multiplier (optional)
32	[CUM]	cost unit multiplier (optional)
36	[RI]	reliability index (optional)
40	[DRM]	development risk factor multiplier (optional)
44	[D]	inside diameter = 0 computed > 0 user specified
48	IVT	valve type = 1 butterfly, pneumatic = 2 butterfly, electric = 3 poppet, 45° stem = 4 poppet, 60° stem = 5 poppet, 90° stem = 6 check, split flapper

Sizing Component VC0MP

Vapor Cycle Compressor

Purpose: This component will compute weight, cost units, reliability index, and development risk factor for a vapor cycle compressor assembly, which includes the compressor, the drive, refrigerant charge, refrigerant lines, and the receiver. The available types of drives are:

- | IDT | Type |
|-----|--------------------|
| 1 | turbine |
| 2 | electric motor, AC |
| 3 | electric motor, DC |
| 4 | hydraulic motor |

Equations:

1. Weight:

For compressor:

$$Wt_{comp} = 0.5 Wt_{drive}$$

For refrigerant charge:

$$Wt_{rc} = \rho V_{rc} / 1728$$

where

$$V_{rc} = 0.3 V_{evap} + 0.6 V_{cond} + 5.184 W V_{in}$$

V_{in} = specific volume at compressor inlet

For receiver:

$$Wt_r = 0.012 V_{rc}$$

For refrigerant lines:

$$Wt_{line} = (Wt_{drive} + Wt_{comp} + Wt_{evap} + Wt_{cond} + Wt_r) \times (0.1 + 0.01 L)$$

2. Cost Units:

For IDT = 1:

$$CU = 5.5 Wt_{comp}$$

For IDT = 2 & 3:

$$CU = 34.7 + 17.3 (Wt_{comp} + Wt_{drive})$$

For IDT = 4

$$CU = 17.0 + 4.88 (Wt_{comp} + Wt_{drive})$$

3. Reliability Index:

For IDT = 1:

$$RI = 0.02576$$

For IDT = 2 & 3:

$$RI = 0.00873$$

For IDT = 4:

$$RI = 0.02576$$

4. Development Risk Factor:

For IDT = 1:

$$DR = 1.15$$

For IDT = 2, 3, & 4:

$$DR = 1.0$$

5. Weight Standard Error:

For IDT = 1:

$$Wt_d = [(0.164 Wt_{comp})^2 + (0.12 Wt_r)^2 + (0.12 Wt_{rc})^2$$

$$+ (0.12 Wt_{line})^2]^{1/2}$$

For IDT = 2:

$$Wt_d = [(0.1665 Wt_{drive})^2 + (0.12 Wt_r)^2 + (0.12 Wt_{rc})^2$$

$$+ (0.12 Wt_{line})^2]^{1/2}$$

For IDT = 3:

$$Wt_d = [(0.388 Wt_{drive})^2 + (0.12 Wt_r)^2 + (0.12 Wt_{rc})^2$$

$$+ (0.12 Wt_{line})^2]^{1/2}$$

For IDT = 4:

$$Wt_d = [(0.3 Wt_{drive})^2 + (0.12 Wt_r)^2 + (0.12 Wt_{rc})^2$$

$$+ (0.12 Wt_{line})^2]^{1/2}$$

6. Installation Weight Factor:

$$Wt_i = 0.205 (Wt_{drive} + Wt_{comp} + Wt_r)$$

7. Volume:

For IDT = 1

$$V = 0.2 D_{turb}^4$$

For IDT = 2 & 3:

$$V = 10 (Wt_{drive} + Wt_{comp})$$

For IDT = 4

$$V = 20 (Wt_{drive} + Wt_{comp})$$

8. Compressor Horsepower:

$$HP = 0.02356 W (H_{out} - H_{in})$$

9. Compressor Efficiency:

$$\eta = (H_s - H_{in}) / (H_{out} - H_{in})$$

Restriction:

1. This component can only be used for fluid type 3.

Notes:

1. Refer to sizing subroutine COMP section for equations pertinent to drives.
2. The component printout consists of weight, cost units, reliability index, development risk factor, and the following items:

W - flow rate

PI - inlet pressure

P_O - outlet pressure

TI - inlet temperature

T_O - outlet temperature

HI - inlet humidity

HO - outlet humidity

FT - fluid type

FN - fluid properties table relative number

WTCD - weight of compressor with drive

WTR - weight of receiver

WTRC - weight of refrigerant charge

WTL - weight of refrigerant lines

V_{OL} - volume of compressor with drive

N - shaft speed

EFF - compressor efficiency

HP - compressor fluid horsepower

Additional printout consists of drive type and the following items:

EFF - drive efficiency

HP - drive input horsepower

WT - drive weight

3. The refrigerant charge density is evaluated at 15°F below the compressor inlet temperature.
4. The value of volume is stored in general argument 71.

Sizing Component VCOMP

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-5	CN	VCOMP component name
8	CC	change code
12	NC	card number
16	NL	leg number
20	NSI	inlet station number
24	NSO	outlet station number
28	NST	shaft number
32	[WTM]	weight multiplier (optional)
36	[CUM]	cost unit multiplier (optional)
40	[RI]	reliability index (optional)
44	[DRM]	development risk factor multiplier (optional)
48	[EV]	evaporator core volume
52	[CV]	condenser core volume
56	IDT	types of drives = 1 turbine = 2 electric motor, AC = 3 electric motor, DC = 4 hydraulic motor
60	[D]	turbine wheel diameter (not required if IDT ≠ 1)
64	-	
68	[L]	line length between evaporator and condenser
72	[CWT]	condenser weight
76	[EWT]	evaporator weight

Sizing Component WSEP
Water Separator

Purpose: This component will compute weight, cost units, reliability index, development risk factor, and volume for water separators.

Equations:

1. Weight:

$$Wt = 0.0936 W$$

2. Cost Units:

$$CU = 18.82 + 2.08 Wt$$

3. Reliability Index:

$$RI = 0.00285$$

4. Development Risk Factor:

$$DR = 1.0$$

5. Weight Standard Error:

$$Wt_d = 0.173 Wt$$

6. Installation Weight Factor:

$$Wt_i = 0.205 Wt$$

7. Volume:

$$V = 2.11 W^{1.5}$$

Restrictions:

1. This component can only be used for fluid type 2.

Notes:

1. The component printout consists of weight, cost units, reliability index, development risk factor, and the following items:

W - flow rate

PI - inlet pressure

PØ - outlet pressure

TI - inlet temperature

TØ - outlet temperature

HI - inlet humidity

HØ - outlet humidity

FT - fluid type

FN - fluid properties table relative number

VØL - volume

2. The value of volume is stored in general argument 71.

Sizing Component WSEP

Data Card Format

<u>Card Column</u>	<u>Symbol</u>	<u>Value</u>
1-4	CN	WSEP component name
8	CC	change code
12	NC	card number
16	NL	leg number
20	NSI	inlet station number
24	NSO	outlet station number
28	[WTM]	weight multiplier (optional)
32	[CUM]	cost unit multiplier (optional)
36	[RI]	reliability index (optional)
40	[DRM]	development risk factor multiplier (optional)