# Linear Quadratic Regulator (LQR)

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### 목 차

LQR Problem

Fast Response vs. Low Control Effort

Optimal Root Locus

Robustness of LQR

LQR + Integral Control

Simulation Using Simulink





#### State Feedback Control Law

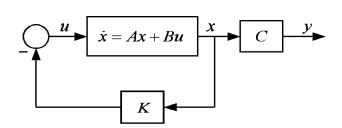
Control-law design

$$u = -Kx = -\begin{bmatrix} k_1 & k_2 & \cdots & k_n \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \end{bmatrix}$$
$$\dot{x} = Ax + Bu$$
$$\Rightarrow \dot{x} = (A - BK)x$$

Picking the gains K so that λ (A-BK) are in desirable locations

Reference input is set to **zero** at this time

How to design K?



$$G_o = C(sI - A)^{-1}B$$

$$G_q = K(sI - A)^{-1}B$$

$$G_{cl} = \frac{G(s)}{1 + G(s)}$$





### **Selection of Pole Locations**

Dominant Second-Order Poles Method

Linear Quadratic Regulator (LQR) Design

System speed vs. Control effort

For the faster system response, the bigger control effort is necessary How to balance between the fast response and low control effort?

The simplified version of LQR problem

Performance index (cost function)

$$J = \int_0^\infty [z^2(t) + \rho u^2(t)]dt$$

ρ: weighting factor

What is the control law that minimize J for

$$\dot{x} = Ax + Bu$$
  $z = C_1 x$ 

z: output or tracking error to be minimized





## Simplified version of LQR

Linear Quadratic Regulator (LQR) Design

Choosing different values of  $\rho$  can provide us with pole locations that achieve varying balances between a **fast response** and a **low** control effort.

 $J = \int_0^\infty [z^2(t) + \rho u^2(t)] dt$ 

#### Small p

- → Low penalty on the control usage
- $\rightarrow$  Small values of  $\int_0^\infty z^2(t)dt$
- → Faster response

#### Large p

- → High penalty on the control usage
- $\rightarrow$  Small values of  $\int_0^\infty u^2(t)dt$
- → Low control effort

Optimal control law is given by linear state feedback u=-Kx

K: determined from Riccati equation





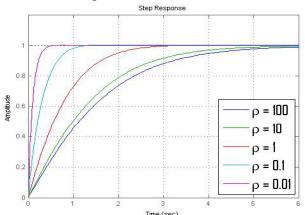
# Simplified version of LQR



Simple Inverted Pendulum  $A = \begin{bmatrix} 0 & 1 \\ 1 & 0 \end{bmatrix}$ ,  $B = \begin{bmatrix} 0 \\ -1 \end{bmatrix}$ ,  $C = \begin{bmatrix} 2 & 1 \end{bmatrix}$ , D = 0

$$z = y$$
 and " $\rho = 1$ " Case  $\rightarrow$  K=[-3.24 -2.73]

ρ	K	λ
100	-2.0198	-1.0062 + 0.0860i
	-2.0124	-1.0062 - 0.0860i
10	-2.1832	-1.0567 + 0.2581i
	-2.1134	-1.0567 - 0.2581i
1	-3.2361	-1.3668 + 0.6067i
	-2.7335	-1.3668 - 0.6067i
0.1	-7.4031	-2.4903 + 0.4490i
	-4.9806	-2.4903 - 0.4490i
0.01	-21.0250	-2.0238
	-11.9185	-9.8947



Comparison between LQR and pole placement

LQR provides guaranteed stability margins, while the stability margins of state feedback pole placement technique are not known





#### **Preliminaries**

The sol. Of  $\dot{x} = Fx$ :  $x = e^{Ft}x_0$ 

A matix is symmetric if  $A^T = A$ 

A symmetric matrix Q is positive (semi)definite if

$$x^TQx > 0 (x^TQx \ge 0), \ \forall x \in \mathbb{R}^n$$

- If (A, B) is controllable (*stabilizable*), ∃ K s.t. all the eigenvalues of (A BK) can be placed at any position (*LHP*) of the complex plane.
- If (C, A) is observable (detectable),  $\exists$  L s.t. all the eigenvalues of (A LC) can be placed at any position (LHP) of the complex plane.





## LQR Design

Linear Quadratic Regulator (LQR) problem

$$\dot{x} = Ax + Bu$$

$$y = Cx$$

$$J = \int_0^\infty (x^T Q x + u^T R u) dt$$

- Q, R: weighting matrix (Q: p.s.d. R: p.d.  $\rightarrow$  J ≥ 0)
- For J to achieve its min. value, both terms must go to zero.

$$J = \int_0^\infty x^T (Q + K^T R K) x dt \qquad u = -Kx$$

$$= x_0^T \left( \int_0^\infty e^{F^T t} S e^{F t} dt \right) x_0 \qquad F = A - BK$$

$$=: x_0^T P x_0, P \ge 0. \qquad x = e^{F t} x_0$$





## ARE for LQR Design

Controller Design: algebraic Riccati equation (ARE)

$$A^{T}P + PA + Q - PBR^{-1}B^{T}P = 0$$
  
$$u = -Kx, \quad K = R^{-1}B^{T}P$$

Matlab : K = Iqr(A,B,Q,R)

When  $Q = D^T D$ , the closed-loop matrix F is stable if (D, A) is detectable.

When  $Q = D^T D$ , the matrix P is positive definite if (D, A) is observable.





# Selection of Q and R

Bryson's rule

$$J = \int_0^\infty (x^T Q x + u^T R u) dt$$

Initial choice of Diagonal matrices Q & н

 $Q_{ii} = 1/\text{max}$ . acceptable value of  $[x_i^2]$ 

 $R_{ii} = 1/\text{max}$ . acceptable value of  $[u_i^2]$ 

Symmetric Root Locus

$$J = \int_0^\infty (\underbrace{x^T C^T C x}_{y^T y} + u^T R u) dt$$

$$A^T P + P A + C^T C - P B R^{-1} B^T P = 0$$

$$K = R^{-1} B^T P$$





## Symmetric Root Locus

$$G_{o} = C(sI - A)^{-1}B$$

$$G_{q} = K(sI - A)^{-1}B$$

$$G_{q} = K(sI - A)^{-1}B$$

$$G_{q} = K(sI - A)^{-1}B$$

$$C^{T}C = (-sI - A^{T})P + P(sI - A) + PBR^{-1}B^{T}P$$

$$B^{T}(-sI - A^{T})^{-1}C^{T}C(sI - A)^{-1}B$$

$$= B^{T}P(sI - A)^{-1}B + B^{T}(-sI - A^{T})^{-1}PB$$

$$+ B^{T}(-sI - A^{T})^{-1}PBR^{-1}B^{T}P(sI - A)^{-1}B$$

$$G_{o}^{T}(-s)G_{o}(s) = RG_{q}(s) + G_{q}^{T}(-s)R + G_{q}^{T}(-s)RG_{q}(s)$$

$$R + G_{o}^{T}(-s)G_{o}(s) = R + RG_{q}(s) + G_{q}^{T}(-s)R + G_{q}^{T}(-s)RG_{q}(s)$$

$$R + G_{o}^{T}(-s)G_{o}(s) = [I + G_{q}^{T}(-s)]R[I + G_{q}(s)]$$

$$1 + \rho^{-1}G_{o}(-s)G_{o}(s) = [1 + G_{q}(-s)][1 + G_{q}(s)]$$

$$1 + \frac{1}{\rho}(-1)^{m-n} \frac{\prod_{(s-z_{i})}^{m}(s+z_{i})}{\prod_{(s-p_{i})}^{n}(s+p_{i})} = 0$$





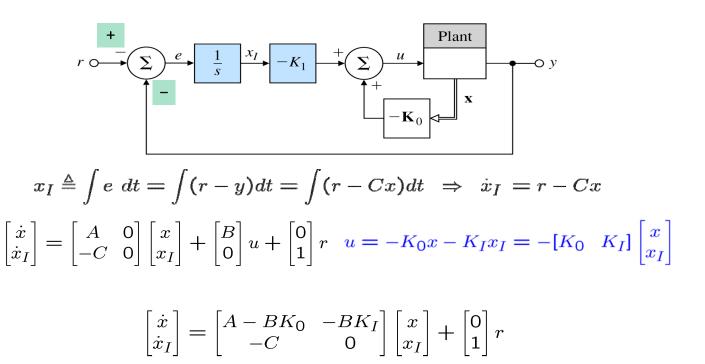
# Symmetric Root Locus Ex.



$$A = \begin{bmatrix} 0 & 1 \\ 1 & 0 \end{bmatrix}, B = \begin{bmatrix} 0 \\ -1 \end{bmatrix}, C = \begin{bmatrix} 2 & 1 \end{bmatrix}, D = 0$$



# **LQR + Integral Control**



16

## Matlab Simulation (LQR+Integral)

```
 A = \begin{bmatrix} 0 & 1 \\ 1 & 0 \end{bmatrix}, \ B = \begin{bmatrix} 0 \\ -1 \end{bmatrix}, \ C = \begin{bmatrix} 2 & 1 \end{bmatrix}, \ D = 0  (Simple Inverted Pendulum) 
 % Sec.2-3 / p.19 / LQR + Integral clear, clc 
  A = \begin{bmatrix} 0 & 1; \ 1 & 0 \end{bmatrix}; \ B = \begin{bmatrix} 0; \ -1 \end{bmatrix}; \ C = \begin{bmatrix} 2 & 1 \end{bmatrix}; \ D = 0;   Aa = \begin{bmatrix} A \ zeros(2,1); \ -C \ 0 \end{bmatrix}; \ Ba = \begin{bmatrix} B; \ 0 \end{bmatrix}; \ Ca = \begin{bmatrix} C \ 0; \ zeros(1,2) \ 1 \end{bmatrix}; \ Da = \begin{bmatrix} 0 \ 0 \end{bmatrix};   rho = 1;   Q = Ca' * Ca; \ R = rho;   K = \begin{bmatrix} 1qr(Aa, Ba, Q, R) \\ 1ambda = eig(Aa - Ba * K), \\ syscl = ss(Aa - Ba * K), \end{bmatrix}   syscl = ss(Aa - Ba * K),   syscl = ss(Aa - Ba * K),   figure(1)   step(syscl), \ grid \ on,
```