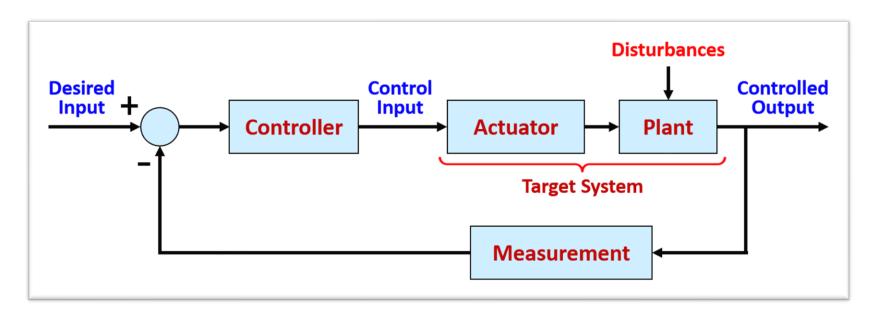
# **Dynamic Response 2**

### Lecture 5:

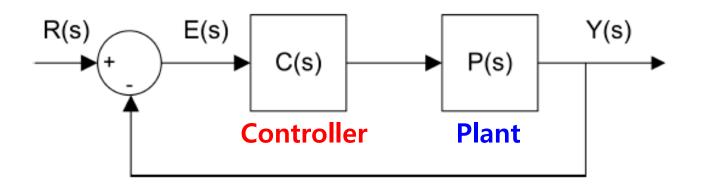
- Pole Placement Design Method
- Stability



#### **Prof. Seunghoon Woo**

Department of Automotive Engineering | College of Automotive Engineering | KOOKMIN UNIVERSITY

### Pole Placement Method



Manual Tuning

- ✓ Time-consuming effort !!

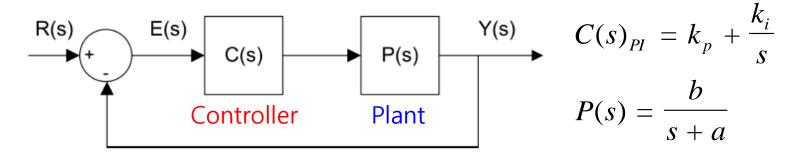
Automatic Tuning Method



- ✓ Indirect PID tuning
- ✓ Still needs fine-tuning effort
- Now, Pole Placement Method!!

Based on the <u>pole locations</u>, the <u>desired characteristic</u> <u>equation of the closed-loop</u> can be produced by controller.

Case 1: First-order System + PI control



The actual characteristic eq. of closed-loop system (PI-control)

$$\frac{Y(s)}{R(s)} = \frac{CP}{1 + CP} = \frac{\frac{k_p s + k_i}{s} \frac{b}{s + a}}{1 + \frac{k_p s + k_i}{s} \frac{b}{s + a}} = \frac{k_p b s + k_i b}{s^2 + (a + k_p b) s + k_i b}$$

The desired characteristic eq. of closed-loop system

$$T(s) = \frac{\omega_n^2}{s^2 + 2\varsigma\omega_n s + \omega_n^2}$$

- Case 1: First-order System + PI control (cont'd)
  - Step 0: Model matching condition

$$s^{2} + (a + k_{p}b)s + k_{i}b = s^{2} + 2\varsigma\omega_{n}s + \omega_{n}^{2}$$
Actual C-L system Desired C-L system

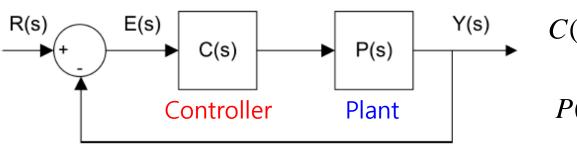
Step response requirement can be satisfied by PI-control gains.

(1) Overshoot 
$$M_p = 1 + \exp\left(-\zeta \omega_n T_p\right) = 1 + \exp\left(-\frac{\pi \zeta}{\sqrt{1-\zeta^2}}\right)$$

(2) Rising time 
$$T_r = \frac{\pi - \varphi}{\omega_d}$$
 where,  $\omega_d = \omega_n \sqrt{1 - \varsigma^2}$ ,  $\cos \varphi = \varsigma$ 

(3) Settling time 
$$T_s \approx \frac{4.6}{\varsigma \omega_n} = \frac{4.6}{\sigma}$$

Case 1: First-order System + PI control (cont'd)



- $C(s)_{PI} = k_p + \frac{k_i}{s}$ 
  - $P(s) = \frac{b}{s+a}$

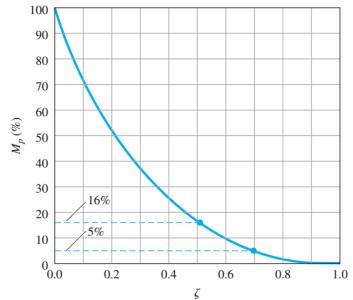
- Step 1: step response requirement
  - (1) Overshoot = 10%

$$M_p = \exp(-\frac{\pi\zeta}{\sqrt{1-\zeta^2}})$$
  $\zeta \approx 0.6$ 

(2) Rising time = 0.1 sec

$$T_{r} = \frac{\pi - \varphi}{\omega_{n} \sqrt{1 - \varsigma^{2}}}$$
where  $\cos \varphi = \varsigma$ 

$$\omega_n = \frac{3.14 - \cos^{-1}(0.6)}{0.1 \times \sqrt{1 - 0.6^2}} = 27.7 \text{ [rad/s]}$$



$$= 27.7 [rad/s]$$

- Case 1: First-order System + PI control (cont'd)
  - Step 2: Check desired pole locations

$$s^{2} + 2\varsigma\omega_{n}s + \omega_{n}^{2} = 0$$

$$\Rightarrow s^{2} + 2(0.6)(27.7)s + 27.7^{2} = 0$$

$$s_{1,2} = -16.6 \pm 22.1j$$

Step 3: Calculate PI gains based on model matching

$$s^{2} + (a + k_{p}b)s + k_{i}b$$

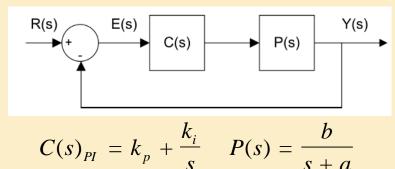
$$= s^{2} + 2\varsigma\omega_{n}s + \omega_{n}^{2}$$

$$s^{1} \longrightarrow k_{p} = \frac{1}{b}(2\zeta\omega_{n} - a)$$

$$k_{i} = \frac{\omega_{n}^{2}}{b}$$

$$\zeta \approx 0.6$$

$$s_{1,2} = -16.6 \pm 22.1j$$



$$a = 30$$

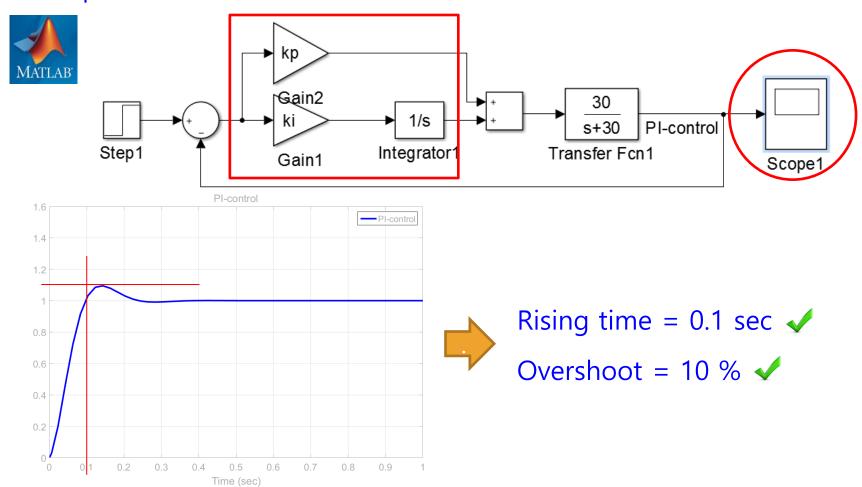
$$k_p = 0.1$$

$$\zeta \approx 0.6$$

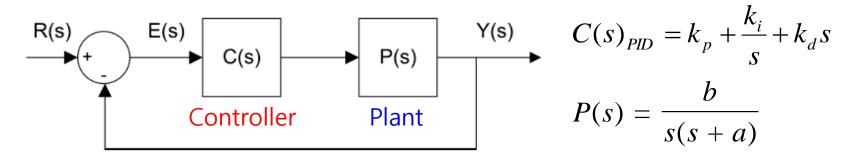
$$k_i = 25.5$$

$$\omega_n = 27.7 [\text{rad/s}]$$

- Case 1: First-order System + PI control (cont'd)
  - Step 4: Simulation model & result



Case 2: Second-order System + PID control



The actual characteristic eq. of closed-loop system (PID-control)

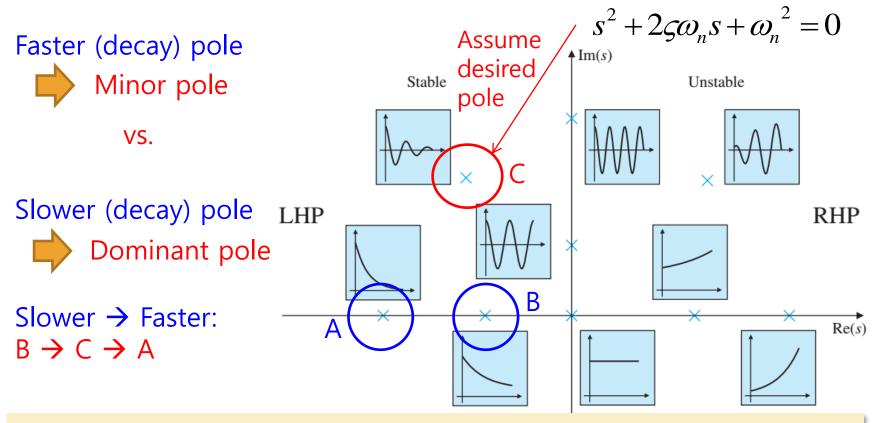
$$\frac{Y(s)}{R(s)} = \frac{CP}{1 + CP} = \frac{\frac{k_d s^2 + k_p s + k_i}{s} \frac{b}{s^2 + as}}{1 + \frac{k_d s^2 + k_p s + k_i}{s} \frac{b}{s^2 + as}} = \frac{b(k_d s^2 + k_p s + k_i)}{s^3 + (bk_d + a)s^2 + (bk_p)s + bk_i}$$

The desired characteristic eq. of closed-loop system

$$T(s) = \frac{\omega_n^2}{s^2 + 2\varsigma\omega_n s + \omega_n^2}$$

Different order !!. How ??

- Case 2: Second-order System + PID control (cont'd)
  - 2<sup>nd</sup>-order desired C-L system vs. 3<sup>rd</sup>-order controlled C-L system
  - So, we need a real pole to make 3<sup>rd</sup>-order desired C-L system!!



Thus, real pole A (faster) does NOT longer affect to desired pole C!!

- Case 2: Second-order System + PID control (cont'd)
  - Step 0: Model matching condition with a minor (faster) pole

$$s^{3} + (bk_{d} + a)s^{2} + (bk_{p})s + bk_{i} = (s + p)(s^{2} + 2\varsigma\omega_{n}s + \omega_{n}^{2})$$
Actual C-L system
$$besired C-L system s_{1,2} = -5.75 \pm 11.4j$$

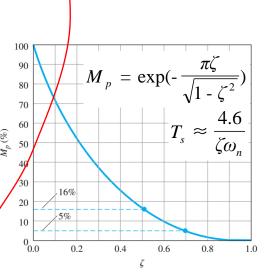
$$= s^{3} + \{(p) + 2\varsigma\omega_{n}\}s^{2} + \{2\varsigma\omega_{n}(p) + \omega_{n}^{2}\}s + \omega_{n}^{2}(p)$$

$$k_{d} = \frac{1}{b}\{(p) + 2\varsigma\omega_{n} - a\}$$

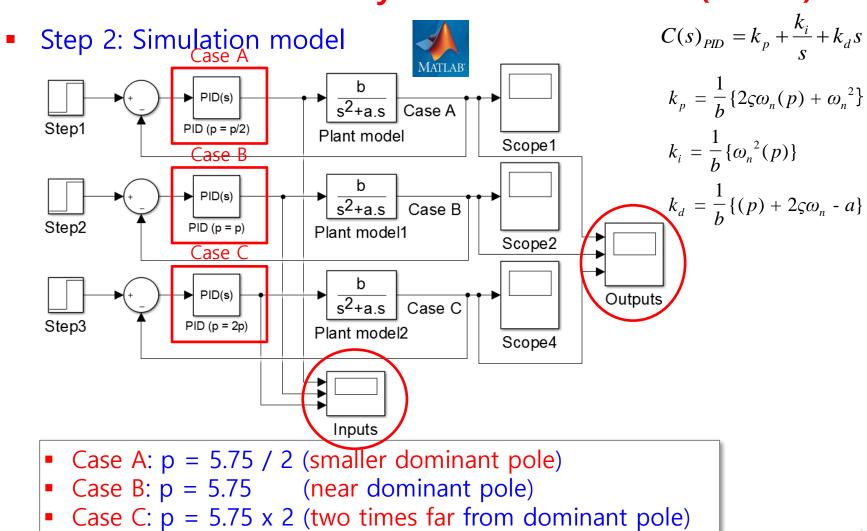
$$a = 10$$

$$b = 10$$

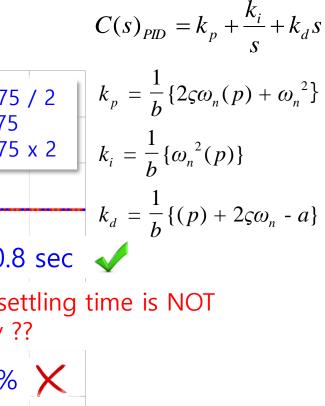
- Step 1: step response requirement
  - (1) Overshoot = 20%  $\rightarrow \zeta \approx 0.45$
  - (2) Settling Time = 0.8 sec  $\longrightarrow$   $\omega_n = 12.8 [rad/s]$



Case 2: Second-order System + PID control (cont'd)



- Case 2: Second-order System + PID control (cont'd)
  - Step 3: Simulation result (1): Outputs





- Case 2: Second-order System + PID control (cont'd)
  - Step 3: Simulation result (1): Outputs (cont'd)

$$C(s)_{PID} = k_p + \frac{k_i}{s} + k_d s$$

Which controller contributes to reduce overshoot ??

$$k_p = \frac{1}{b} \{ 2\varsigma \omega_n(p) + \omega_n^2 \}$$

$$k_i = \frac{1}{b} \{ \omega_n^2(p) \}$$

$$k_d = \frac{1}{b} \{ (p) + 2\varsigma \omega_n - a \} \times \beta$$



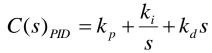
 $\beta > 1$ : increasing damping

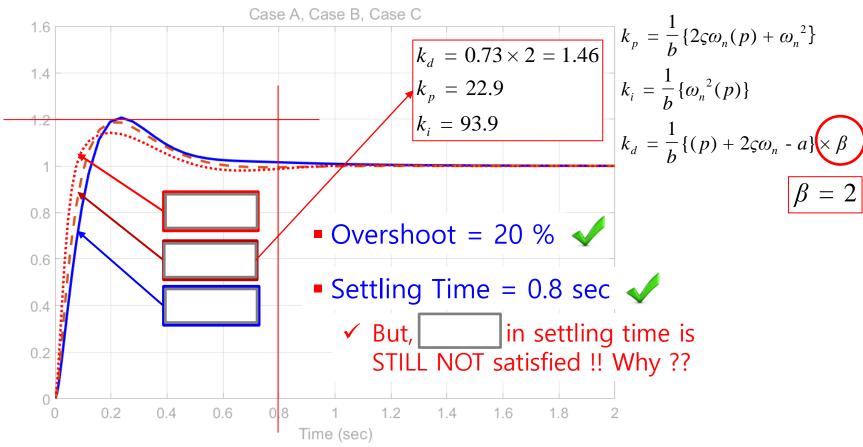
 $\beta$  < 1 : decreasing damping

where,

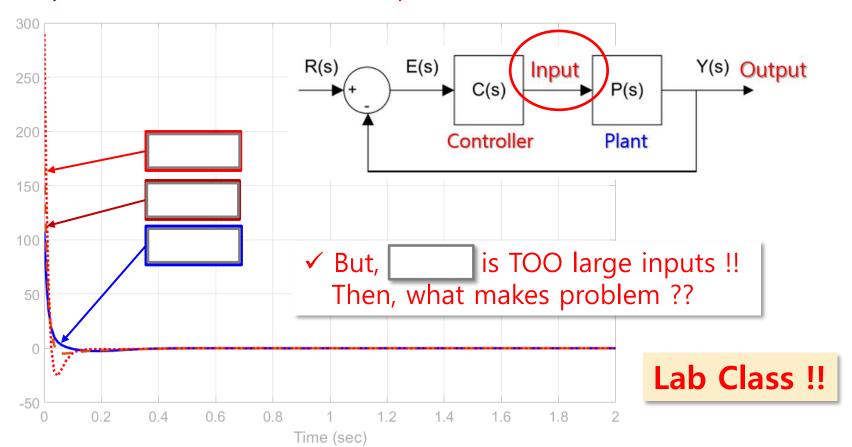
 $\beta$ : weighting factor

- Case 2: Second-order System + PID control (cont'd)
  - Step 3: Simulation result (1): Outputs





- Case 2: Second-order System + PID control (cont'd)
  - Step 3: Simulation result (2): Inputs

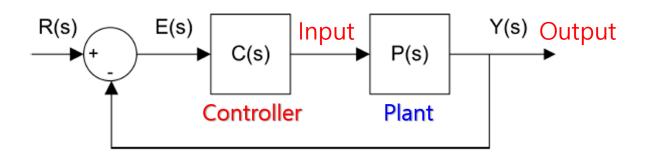


- Case 2: Second-order System + PID control (cont'd)
  - Step 3: Simulation result (2): Inputs (cont'd)

Lab Class !!

For example, DC-motor position control

$$P(s) = \frac{\Theta(s)}{V_a(s)} = \frac{Output}{Input} = \frac{Position}{Voltage} = \frac{K}{s(\tau s + 1)}$$

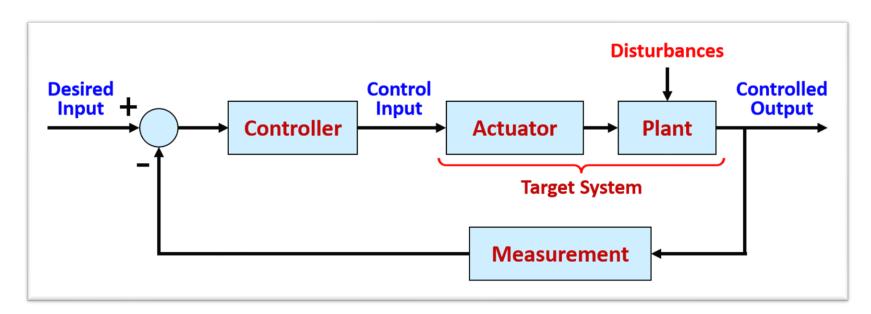


Thus, we should check whether the input capability (or limitation) to perform the controlled output !!

# **Dynamic Response 2**

### Lecture 5:

- Pole Placement Design Method
- Stability

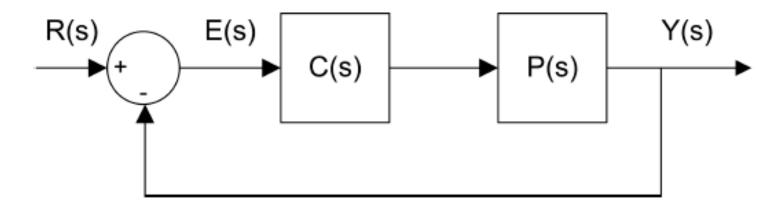


#### **Prof. Seunghoon Woo**

Department of Automotive Engineering | College of Automotive Engineering | KOOKMIN UNIVERSITY

# What is the Stability ??

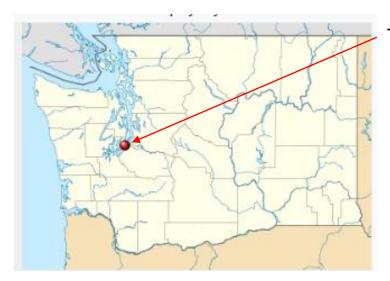
- Utmost important specification in control design !!
- Unstable systems have to be stabilized by feedback.
- Unstable closed-loop systems are useless !!
- What happens if a system is unstable?
  - may hit mechanical/electrical/biological/chemical "stops" (handling with saturation scheme)
  - may break down or burn out



# What happened if a system is unstable?

Tacoma Narrows Bridge (July 1-Nov.7, 1940)

https://www.youtube.com/watch?v=XggxeuFDaDU



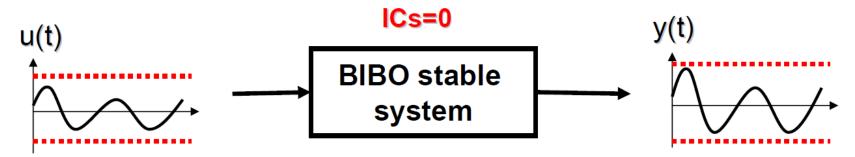
Tacoma city @ Washington State, USA

2008...

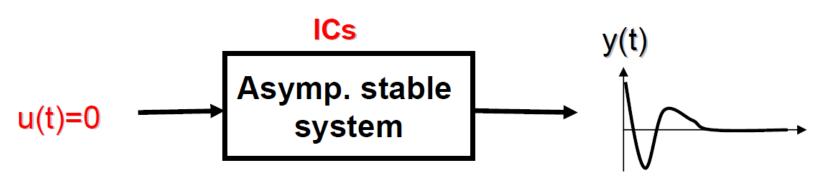


# **Mathematical Definitions of Stability**

BIBO (Bounded-Input-Bounded-Output) stability:
 Any bounded input generates a bounded output.

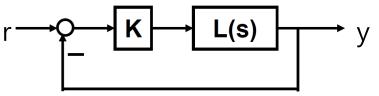


Asymptotic stability:
 Any ICs generates y(t) converging to zero.

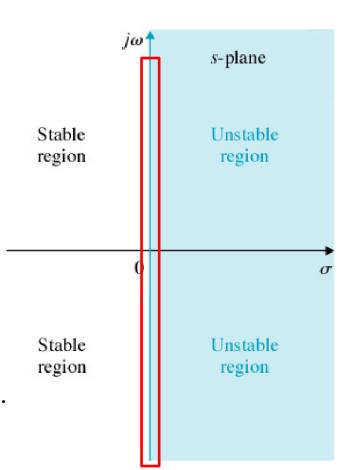


# **Stability Summary**

Let  $s_i$  be poles of rational G(s)G(s) = 1 + KL(s)

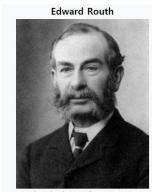


- (BIBO, asymptotically) stable if Re(s<sub>i</sub>)<0 for all i.</li>
- Marginally stable if
  - Re(s<sub>i</sub>)<=0 for all i, and</li>
  - ◆ simple root for Re(s<sub>i</sub>)=0
- Unstable if
  - it is neither stable nor marginally stable.



### **Routh-Hurwitz Criterion**

- This is for LTI systems with a polynomial <u>denominator</u>
  - (without sin, cos, exponential etc.)
- It determines if all the roots of a polynomial
  - lie in the LHP (left half-plane),
  - or equivalently, have negative real parts.
- It also determines the number of roots of a polynomial in the RHP (right half-plane).
- But, it does NOT explicitly compute the roots.



edward John Routh (1831–1907)

orn 20 January 1831
Quebec, Canada

ied 7 June 1907 (aged 76)
Cambridge, England



26 March 1859
Hildesheim, Kingdom of
Hanover (now part of Germany)
d 18 November 1919 (aged 60)
Zürich, Switzerland

# Polynomial and an Assumption

Consider a polynomial of characteristic eq.

$$Q(s) = a_n s^n + a_{n-1} s^{n-1} + \dots + a_1 s + a_0$$

- Assume  $a_0 \neq 0$ 
  - If this assumption does not hold, Q can be factored as

$$Q(s) = s^m \underbrace{(\hat{a}_{n-m}s^{n-m} + \cdots + \hat{a}_1s + \hat{a}_0)}_{\hat{Q}(s)}$$
 where  $\hat{a}_0 \neq 0$ 

• The following method applies to the polynomial  $\hat{Q}(s)$ 

# **Routh Array**

$$Q(s) = a_{n}s^{n} + a_{n-1}s^{n-1} + \cdots + a_{1}s + a_{0}$$
Highest  $\rightarrow s^{n}$ 

$$a_{n} \quad a_{n-2} \quad a_{n-4} \quad a_{n-6} \quad \cdots$$

$$a_{n-1} \quad a_{n-3} \quad a_{n-5} \quad a_{n-7} \quad \cdots$$

$$s^{n-2} \quad b_{1} \quad b_{2} \quad b_{3} \quad b_{4} \quad \cdots$$

$$s^{n-3} \quad c_{1} \quad c_{2} \quad c_{3} \quad c_{4} \quad \cdots$$

$$\vdots \quad \vdots \quad \vdots$$

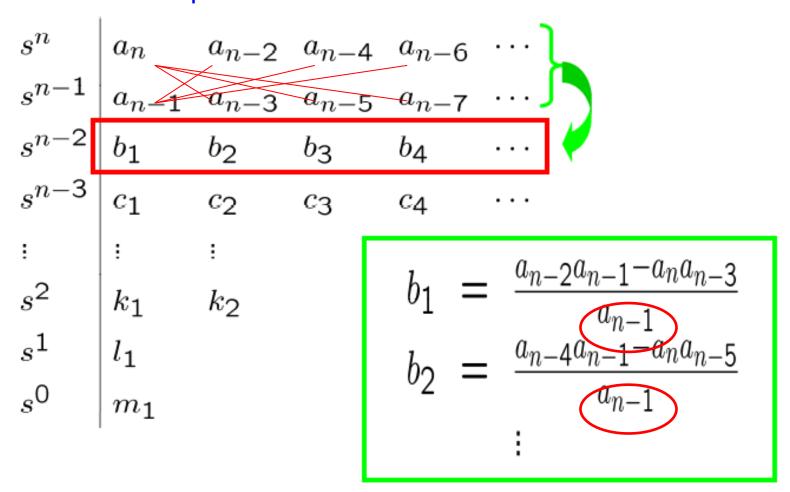
$$s^{2} \quad k_{1} \quad k_{2}$$

$$s^{1} \quad l_{1}$$

$$s^{0} \quad m_{1}$$
From the given polynomial

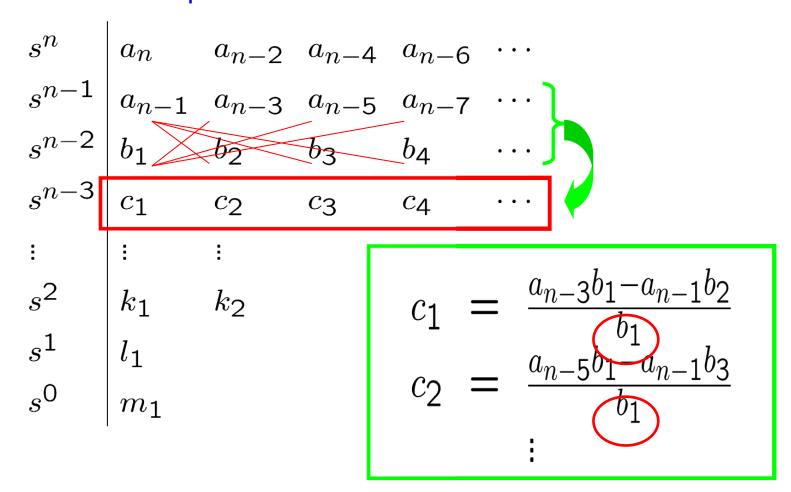
# Routh Array (cont'd)

How to compute the third row

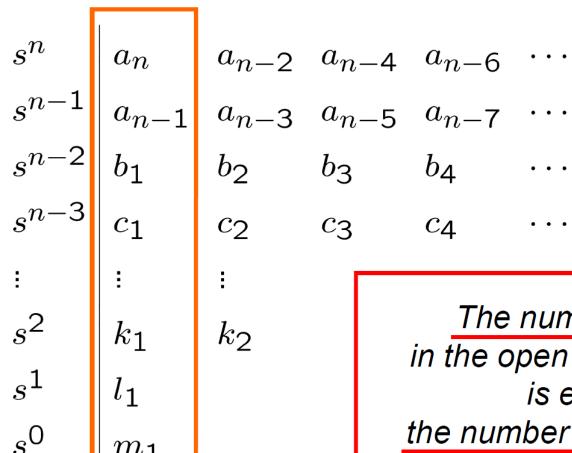


# Routh Array (cont'd)

How to compute the forth row



### **Routh-Hurwitz Criterion**

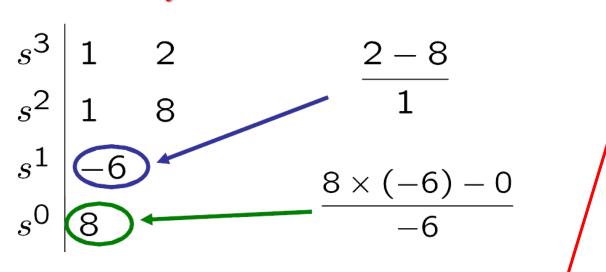


The number of roots
in the open right half-plane
is equal to
the number of sign changes
in the first column of Routh array.

# **Example 1: Simple Polynomial**

$$Q(s) = s^3 + s^2 + 2s + 8 = (s+2)(s^2 - s + 4)$$

### Routh array



### Two sign changes

in the first column

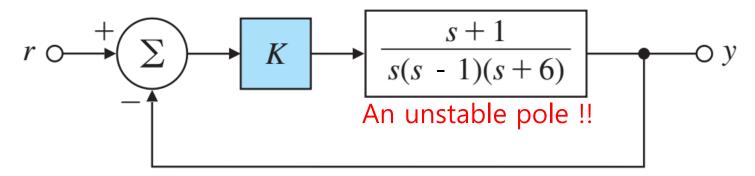
$$1 \rightarrow -6 \rightarrow 8$$



### Two roots in RHP

$$\frac{1}{2} \pm \frac{j\sqrt{15}}{2}$$

# Example 2: In the Feedback System



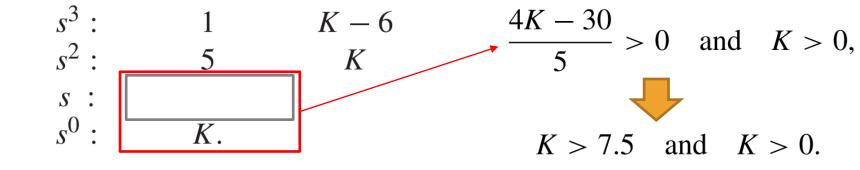
Characteristic eq. is given by

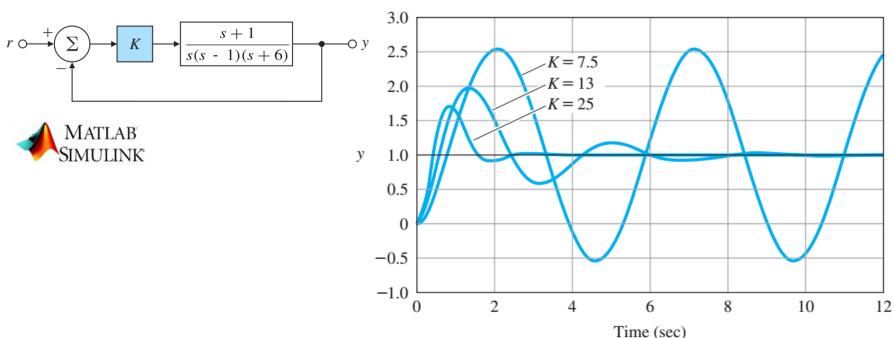
$$1 + K \frac{s+1}{s(s-1)(s+6)} = 0, \quad \Longrightarrow \quad s^3 + 5s^2 + (K-6)s + K = 0.$$

Routh array is 
$$s^3$$
: 1  $K-6$   
 $s^2$ : 5  $K$   
 $s$ :  $(4K-30)/5$   
 $s^0$ :  $K$ .

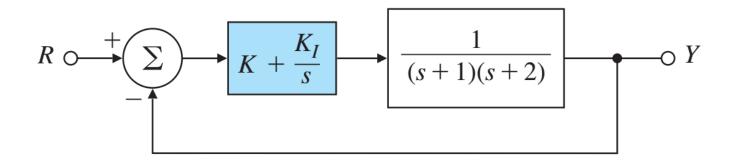
# **Example 2: In the Feedback System (cont'd)**

For the system to be stable, it is necessary that





# **Example 3: PI Controller Design**



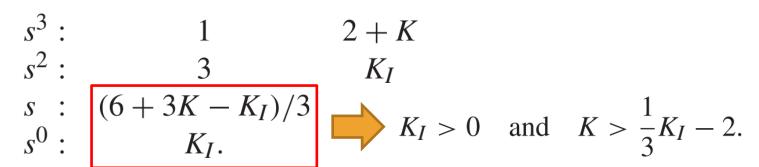
Characteristic eq. is given by

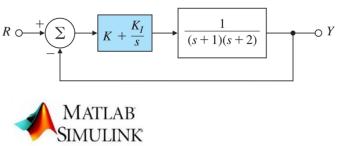
$$1 + \left(K + \frac{K_I}{s}\right) \frac{1}{(s+1)(s+2)} = 0, \quad \Longrightarrow \quad s^3 + 3s^2 + (2+K)s + K_I = 0.$$

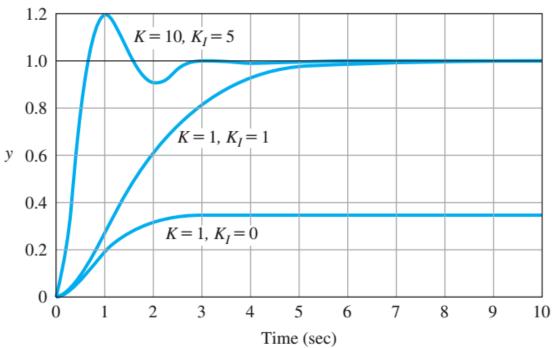
Routh array is 
$$s^3$$
: 1  $2+K$   
 $s^2$ : 3  $K_I$   
 $s$ :  $(6+3K-K_I)/3$   
 $s^0$ :  $K_I$ .

# Example 3: PI Controller Design (cont'd)

For the system to be stable, it is necessary that







# **Special Case I: Routh-Hurwitz Method**

• Question: What if the b<sub>1</sub> (denominator) is zero?

# **Special Case I:** Routh-Hurwitz Method (cont'd)

Consider a polynomial of characteristic eq.

$$Q(s) = s^{5} + 2s^{4} + 3s^{3} + 6s^{2} + 5s + 2 = 0$$

$$\begin{array}{c|c}
\hline
 & & & & \\
 & & & \\
 & s^{5} & & 1 & 3 & 5 \\
\hline
 & s^{4} & & 2 & 6 & 2 \\
\hline
 & s^{3} & & & \\
 & s & & \\
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Then, instead of zero, we can replace with small number ( $\varepsilon = 0$ )

### **\*** Findings:

# **Special Case II: Routh-Hurwitz Method**

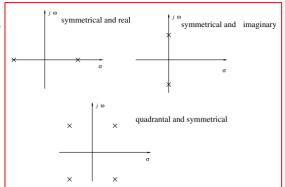
• Question: What if all coefficients in a row are zeros?

# **Special Case II: Routh-Hurwitz Method (cont'd)**

Consider a polynomial of characteristic eq.

$$Q(s) = s^5 + 2s^4 + 4s^3 + 5s^2 + 4s + 2 = 0$$

$S^5$	1	4	4
$s^4$	2	5	2
$s^3$	1.5	3	



$$\begin{pmatrix} s^2 \\ s \end{pmatrix} \begin{pmatrix} 1 \\ 0 \\ 0 \end{pmatrix}$$

 $A(s) = s^2 + 2$ : Auxiliary polynomial equation

It indicates that Q(s) = 0 must have a pair of roots equal magnitude and opposite sign, which are also roots of A(s) = 0.

$s^5$	1	4	4
s <sup>4</sup>	2	5	2
$s^3$	1.5	3	
$s^2$	1	2	$d\Lambda(a)$
S	2	0	$\frac{dA(s)}{ds} = 2s$
1	2	0	ds

❖ Thus, the zero row can be calculated!! → there is no sign change!!

# Summary

### **Summary:**

- PID controller design by using Pole Placement Method
- Stability Criteria
- Routh-Hurwitz Stability Criteria and Control Design Method