Control System Design for Automated Driving

Lecture 05

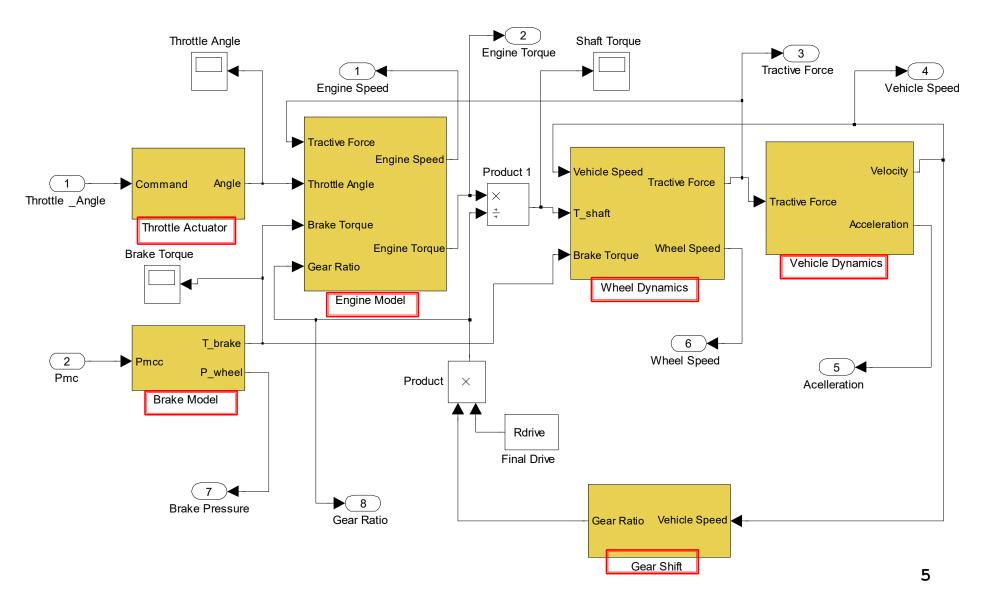






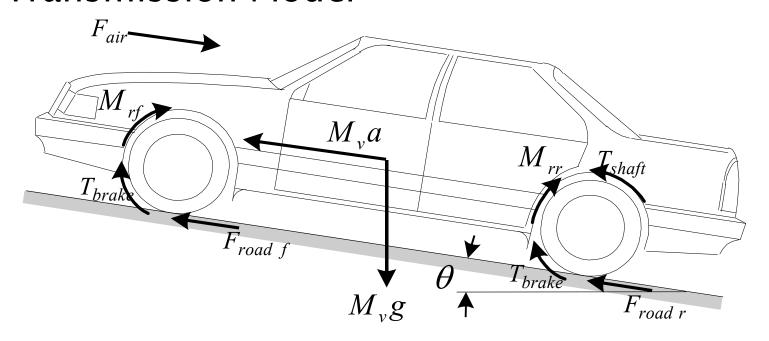
Simulink Model of Vehicle Longitudinal Dynamics

Simulink Model of Vehicle Longitudinal Dynamics



Longitudinal Vehicle & Wheel Dynamics

- Aerodynamic Drag $F_{air} = \frac{\rho}{2} C_d A_f v^2$
- Rolling Resistance $F_{roll} = constant$
- ▶ Road Traction Force F_{road} from Tire Model
- **b** Brake Torque T_{brake} from Brake Model
- Shaft Torque T_{shaft} from Engine & Transmission Model

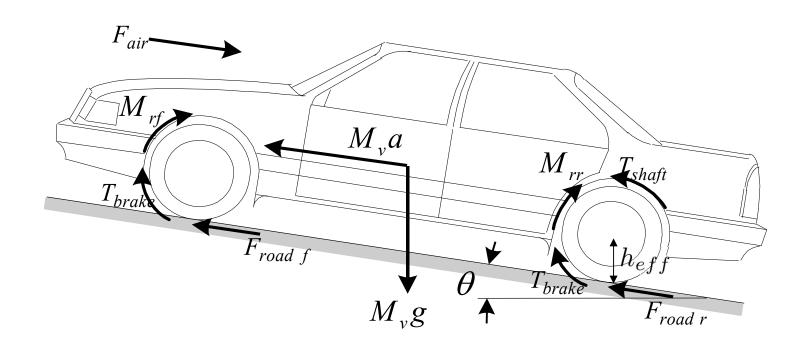


Longitudinal Vehicle & Wheel Dynamics

- Nonlinear Longitudinal Vehicle Dynamics
 - Vehicle Dynamics

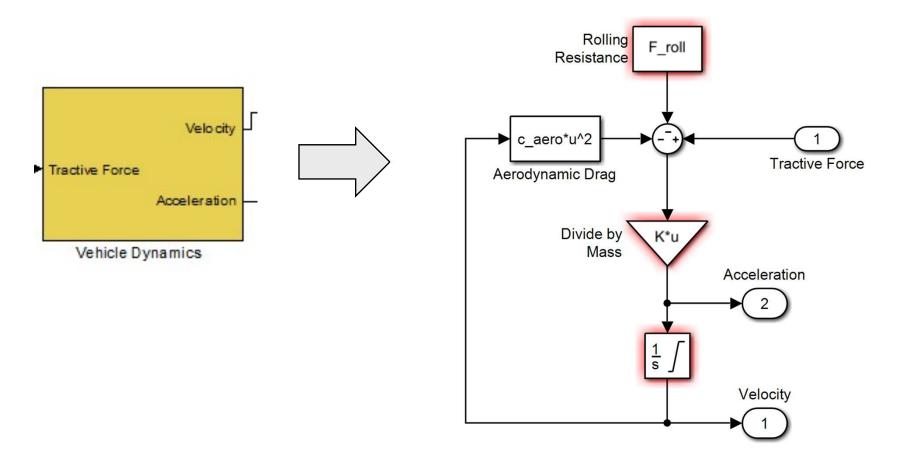
$$M\dot{v} = F_{road} - F_{roll} - F_{air} - Mg\sin\theta$$

$$\circ$$
 Wheel Dynamics $J_{\omega}\dot{\omega}=T_{shaft}-T_{brake}-h_{eff}F_{road}$

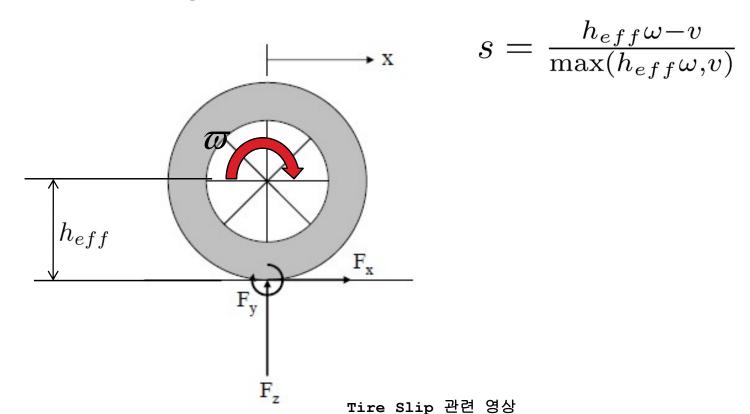


Longitudinal Vehicle Dynamics

Simulink Model of $M\dot{v}=F_{road}-F_{roll}-F_{air}-Mg\sin\theta$ (with $\theta=0$)



- Tire Road Force F_{road} is usually modeled by empirical relationship between slip and force.
- Definition of Longitudinal Tire Slip ratio

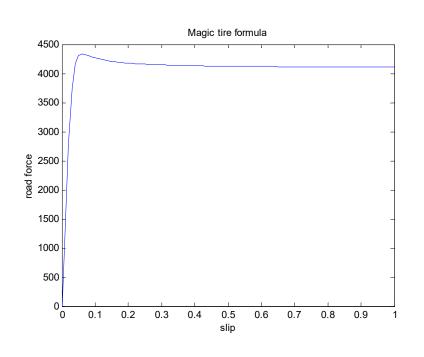


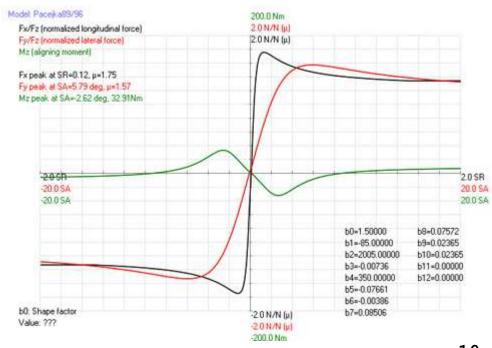
https://www.youtube.com/watch?v=tsnYqCRWTbE9

Bakker-Pacejka 'Magic Tire Formula'

$$F_{road} = D\sin[C\arctan\{B(s+S_h) - E(B(s+S_h) - \arctan(B(s+S_h)))\}]$$

• S_h : bias in the slip curve caused by change in effective tire radii, camber angle and etc.





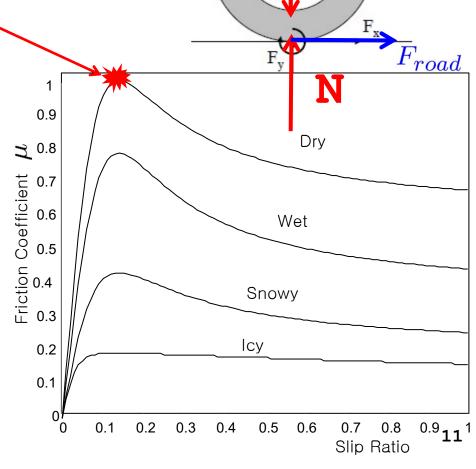
Road Force & μ_{MAX}

Maximum Friction Coefficient

$$F_{road} \le \mu_{\max} N$$

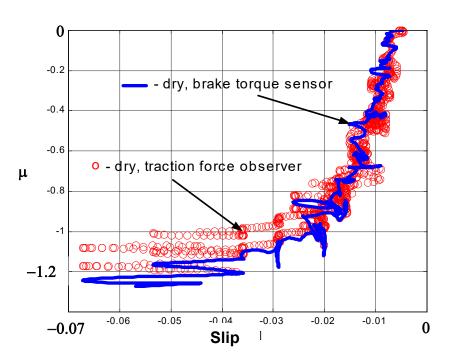
- Road Condition
- Tire Type
- Tread Pattern
- Tread Depth
- Velocity
- Fiction Coefficient Definition as Normalized Force

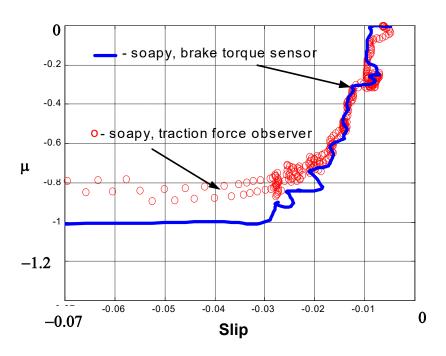
$$\rightarrow \mu = F_{road}/N$$



→ X

Experimental slip curves

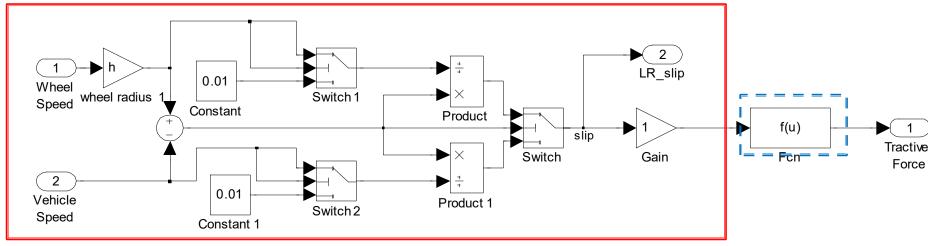




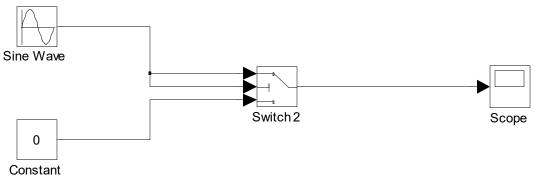
- Simulink Model of Tire
 - Constant block is used to avoid the case of "divide by zero".

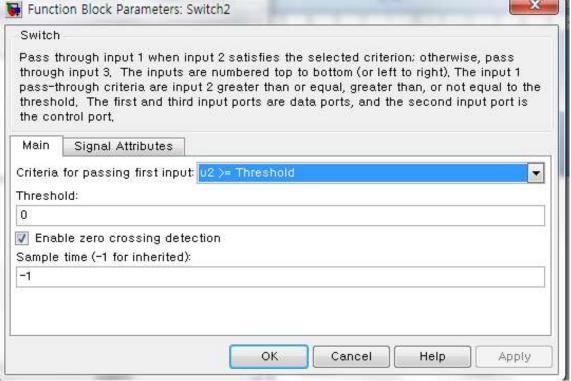
$$s = \frac{h_{eff}\omega - v}{\max(h_{eff}\omega, v)}$$

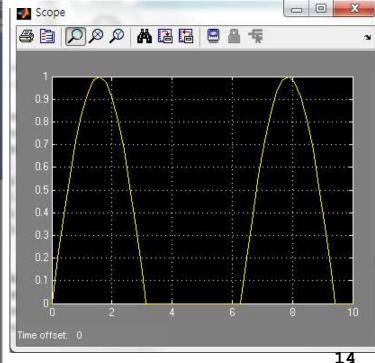
$$F_{road} = D\sin[C\arctan\{B(s+S_h)-E(B(s+S_h)-\arctan(B(s+S_h)))\}]$$







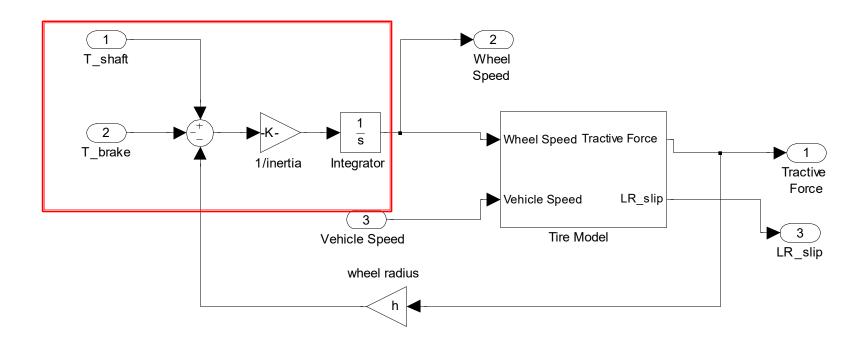




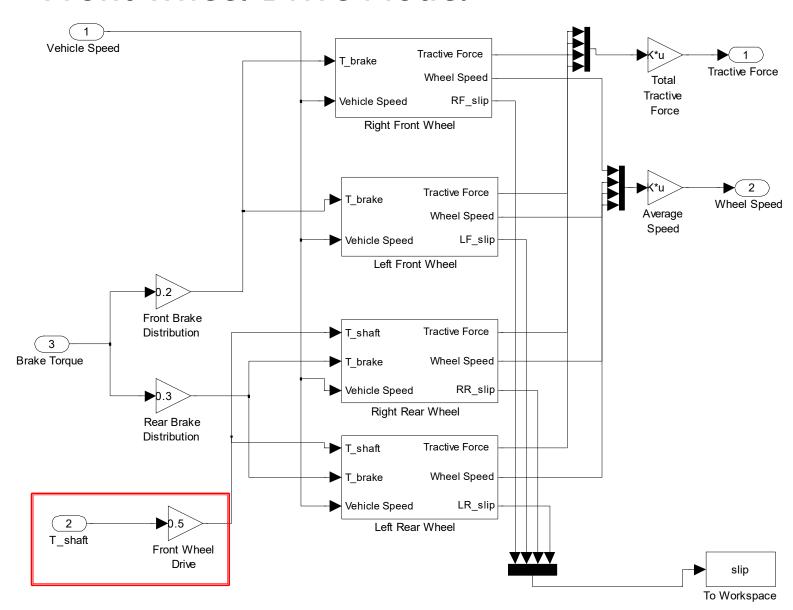
Wheel Dynamics

Wheel Dynamics

$$J_{\omega}\dot{\omega} = T_{shaft} - T_{brake} - h_{eff}F_{road}$$



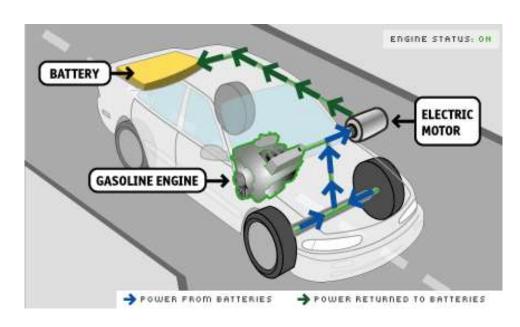
Wheel Dynamics Front Wheel Drive Model



브레이크 제어

- ▶ 브레이크
 - 마찰을 이용하여 차량의 운동에너지를 열에너지로 변환 시켜 감속
 - 최근 Hybrid Electric Vehicle(HEV) 는 회생제동시스템을 이용하여 운동에너지를 전기에너지로 변환시켜 버려지는 에너지를 재사용하여 연비 향상

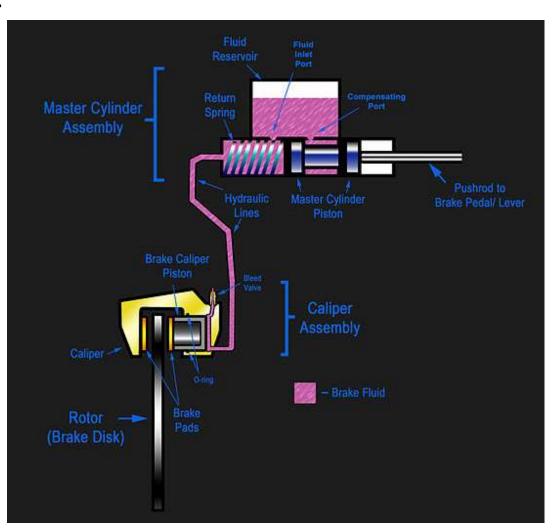




Brake Model

Hydraulic Disc Brake

- ▶ 브레이크 제동력 전달
 - 브레이크 페달
 - → Push rod
 - → 마스터실린더
 - → 브레이크 오일
 - → 브레이크 피스톤
 - → 브레이크 패드
 - → 브레이크 디스크



Brake Model

- Hydraulic Master Cylinder Pressure Model
- Constant Weight Distribution Assumption
- Simple Proportional Model

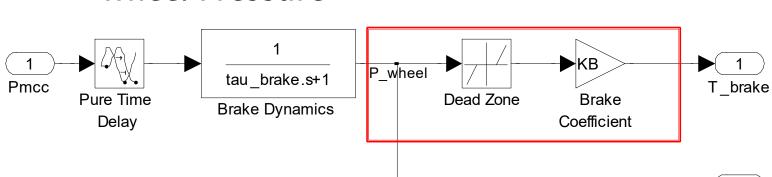
$$T_b = \begin{cases} K_b(P_w - P_{po}), & P_w \ge P_{po} \\ 0, & \text{otherwise} \end{cases}$$

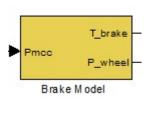


 $P^{P_{po}}$ Pushout Pressure

• *P_{mc}* Master Cylinder Pressure

 \circ P_w Wheel Pressure



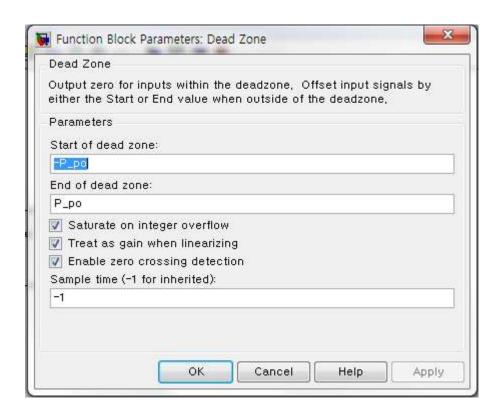


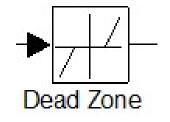


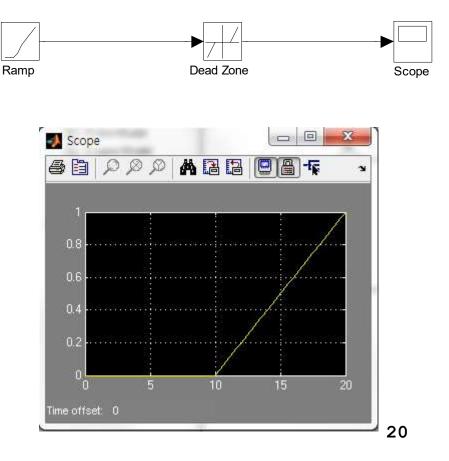
P wheel

Brake Model

- Dead Zone
 - Typical Nonlinear Dynamics
 - Zero the output before the input value is below a certain thresholds





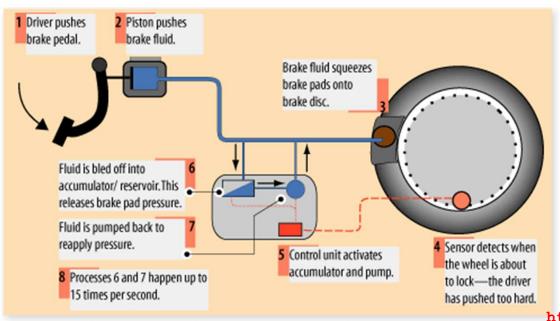


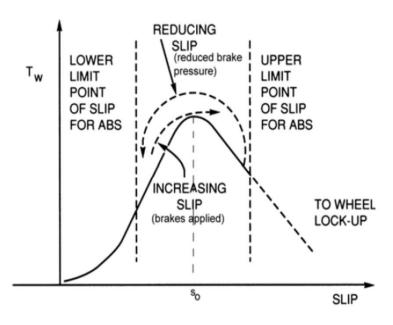
전자식 브레이크 제어

Anti-lock Brake System (ABS)

브레이크에 과도한 힘이 작용하여 브레이크 패드가 디스크를 꽉 물어 타이어가 잠기는 현상을 방지

타이어가 잠기면 자동차는 조향성을 잃고 마찰력이 감소하는 것을 방지





https://www.youtube.com/watch?v=Zg6zufVDqU\

전자식 브레이크 제어

- Electronic Stability Control
 - 브레이크를 통해 <u>횡방향</u> 미끄러짐 을 최소화할 수 있도록 제어
 - Understeer, Oversteer를 최소화 하기 위하여 좌우 브레이크를 다르 게 적용



