



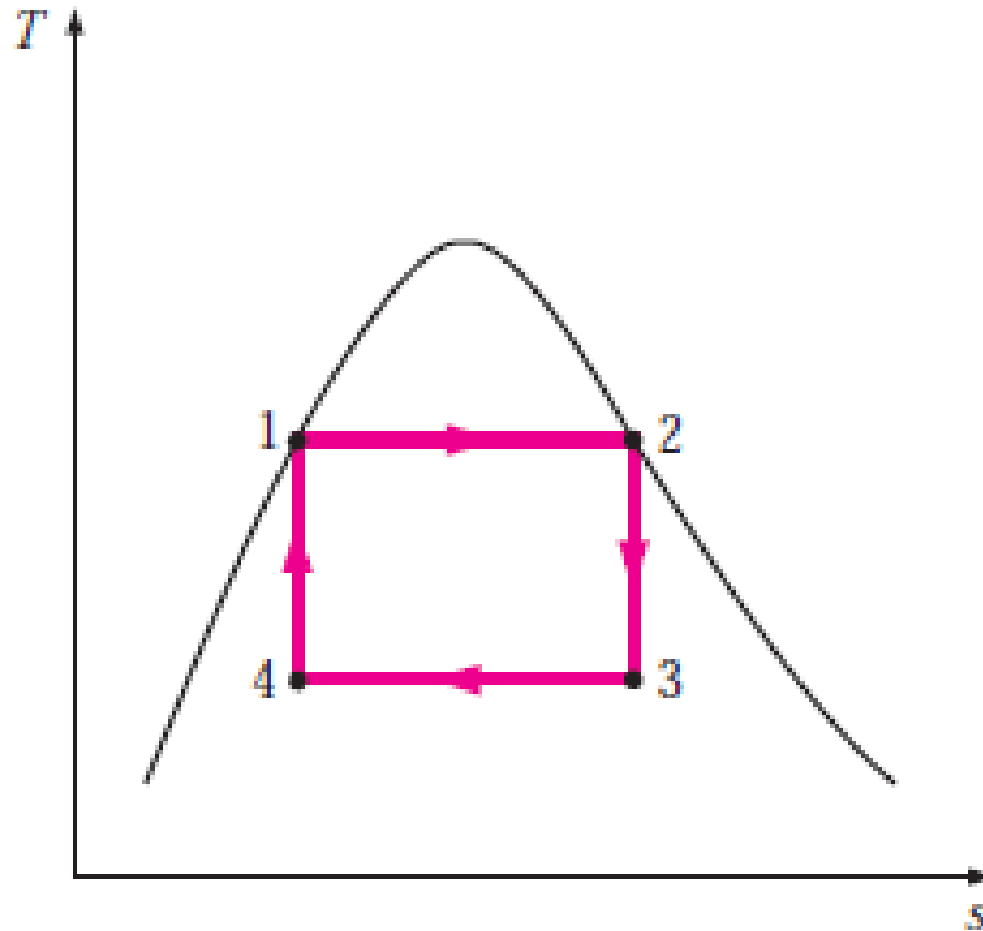
# **ME 266 THERMODYNAMICS 1**

## **- Vapour Power Cycles**

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# VAPOUR POWER CYCLES

## Carnot vapour power cycle



1-2 Constant,  $T$  (isothermal) heat addition

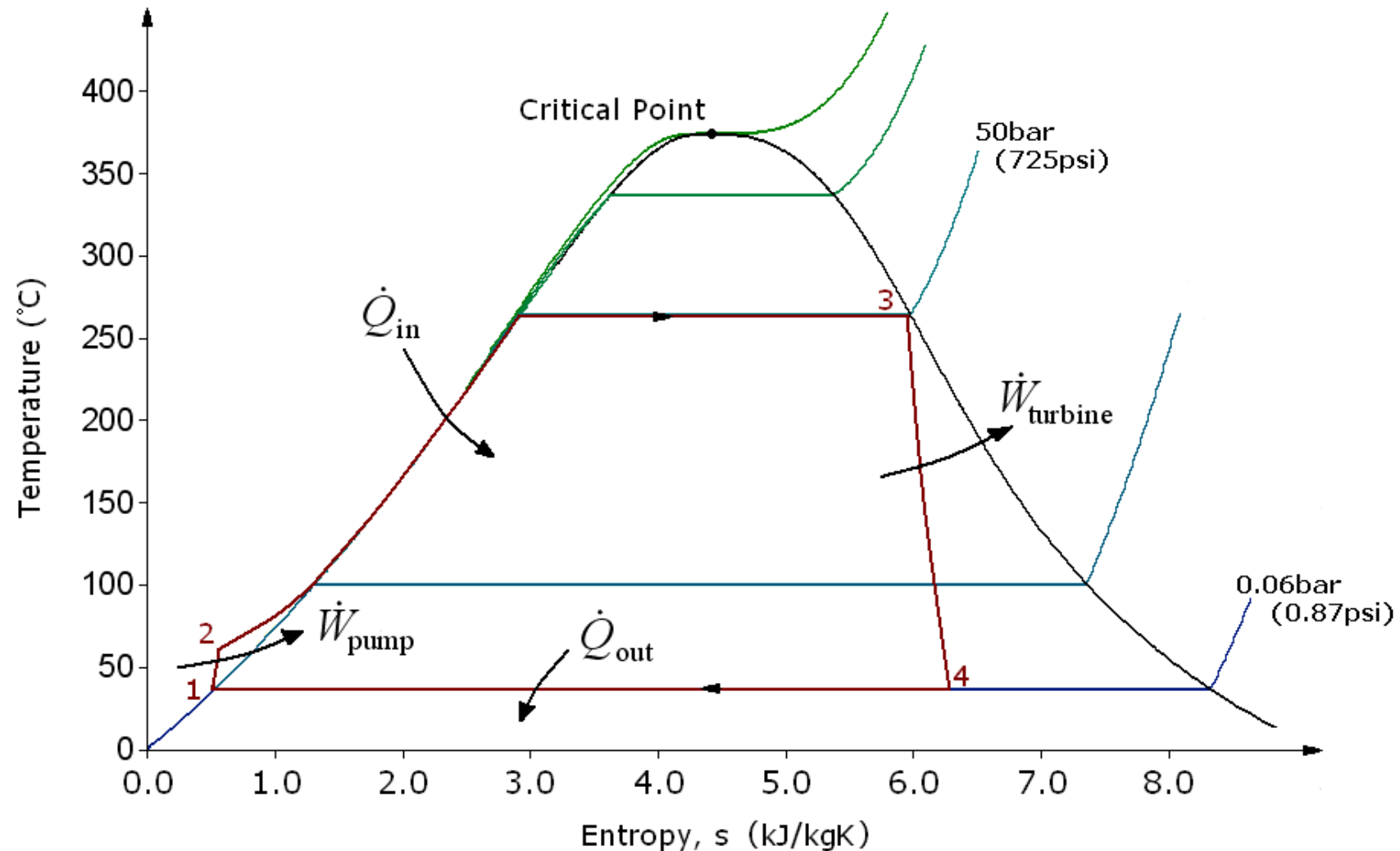
2-3 Constant entropy (isentropic)  
expansion in turbine

3-4 Constant  $P$  const,  $T$  (isothermal)  
heat rejection (condenser)

4-1 Constant entropy (isentropic)  
compression

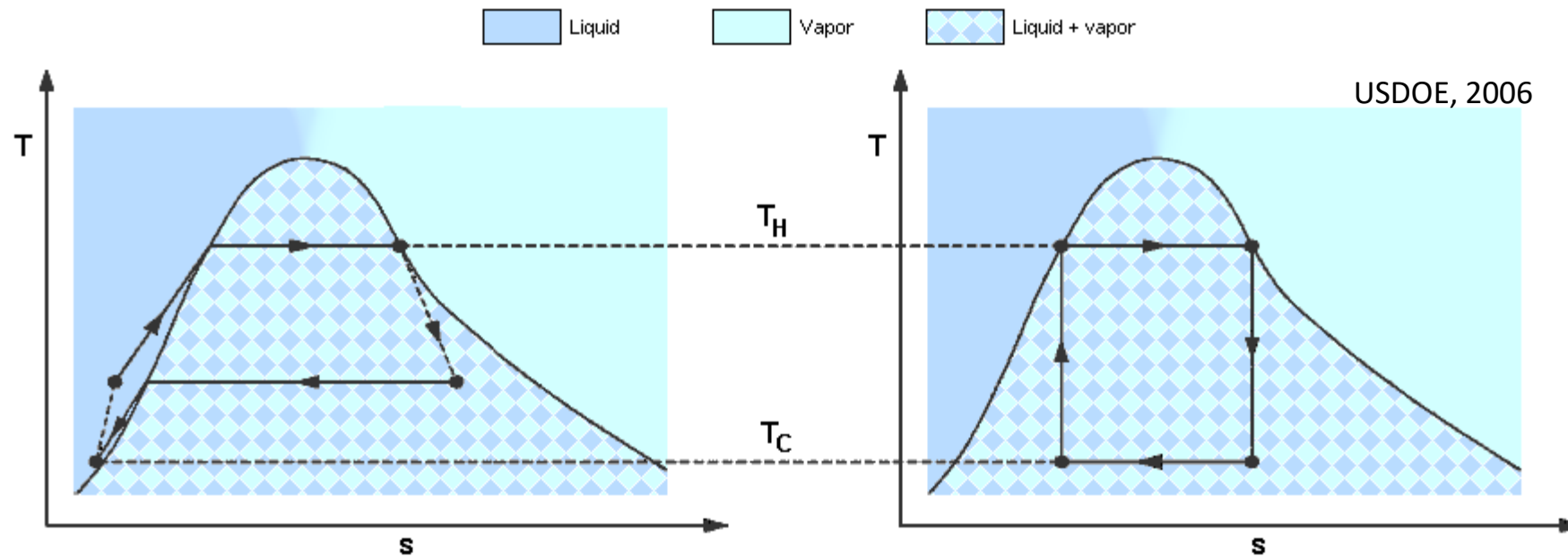
# VAPOUR POWER CYCLES

## Simple Rankine Cycle



# VAPOR POWER CYCLES

## Real vs Carnot

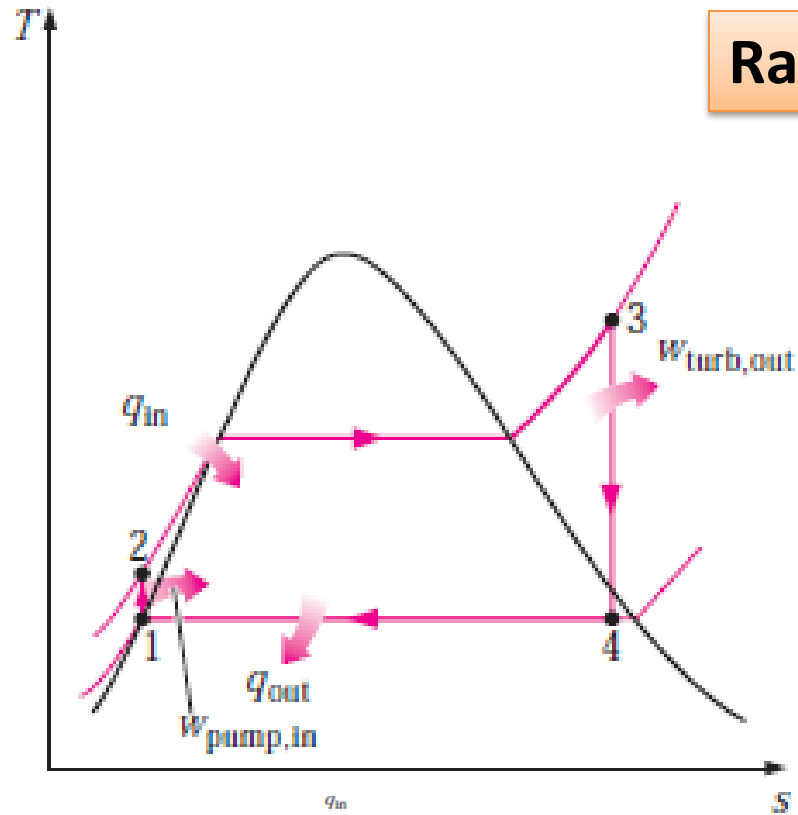


Working fluid is completely condensed and cooled.  
Entropy of real processes increase.

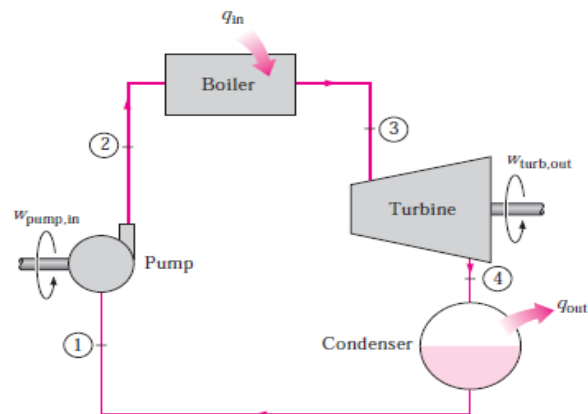


# VAPOR POWER CYCLES

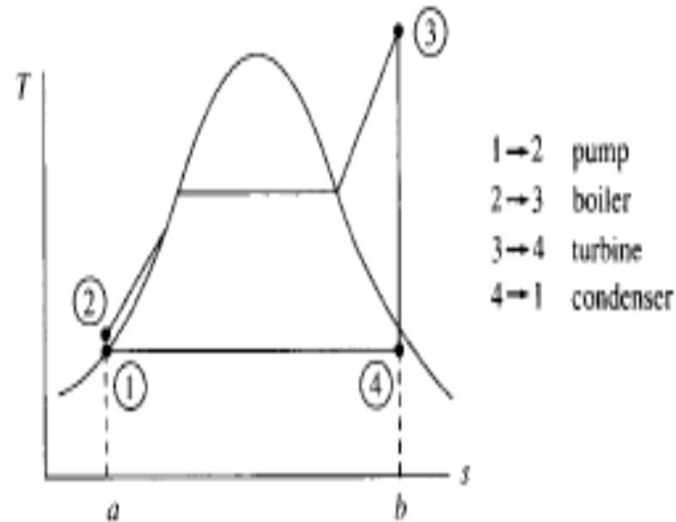
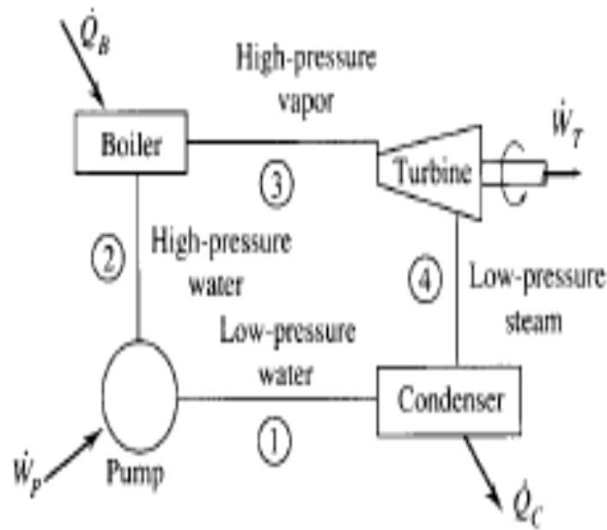
## Rankine cycle with superheat



- 1 → 2: Isentropic compression in a pump*
- 2 → 3: Constant-pressure heat addition in a boiler*
- 3 → 4: Isentropic expansion in a turbine*
- 4 → 1: Constant-pressure heat rejection in a condenser*



# VAPOR POWER CYCLES



- 1→2 pump
- 2→3 boiler
- 3→4 turbine
- 4→1 condenser

$$w_p = v_1(P_2 - P_1)$$

$$q_B = h_3 - h_2$$

$$w_T = h_3 - h_4$$

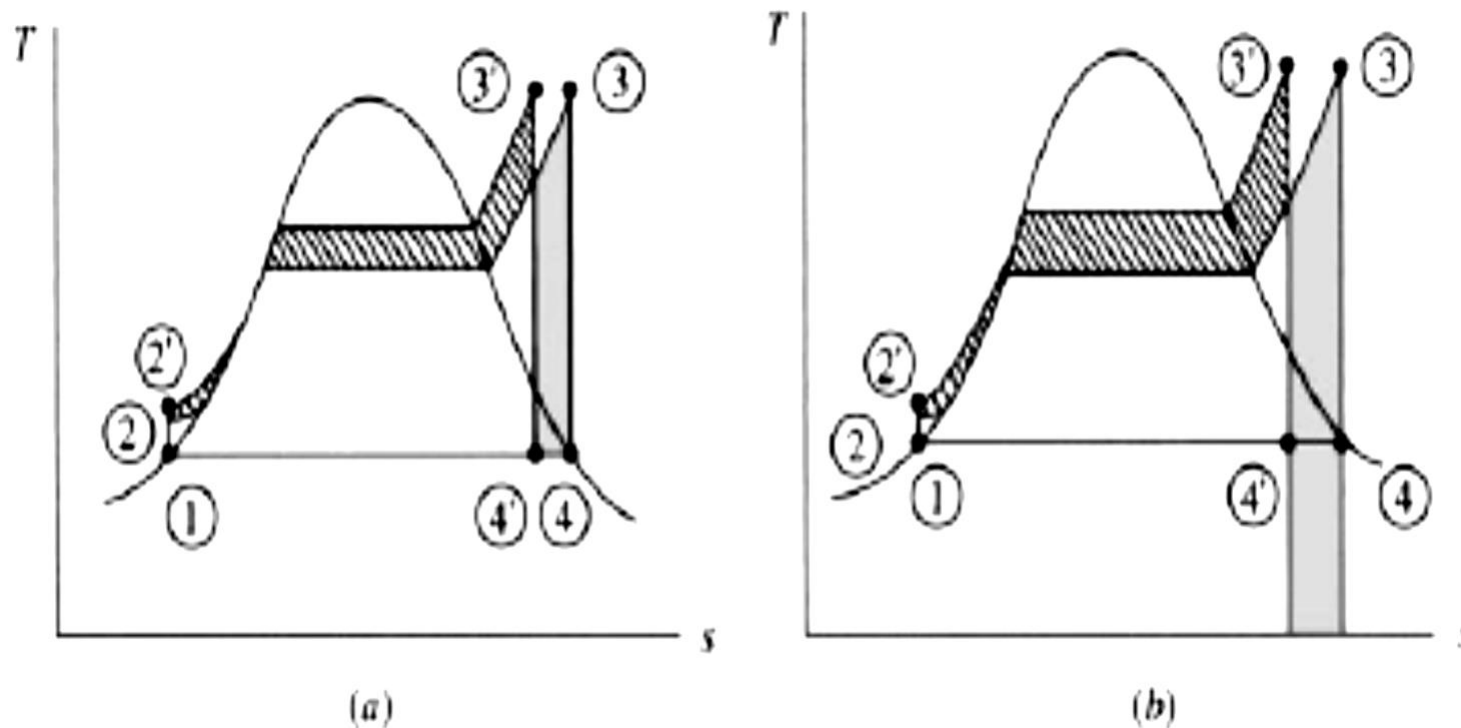
$$q_C = h_4 - h_1$$

$$\eta = \frac{\text{area } 1 - 2 - 3 - 4 - 1}{\text{area } a - 2 - 3 - b - a}$$

$$\eta = \frac{w_T - w_p}{q_B}$$

# IMPROVING THE RANKINE CYCLE EFFICIENCY

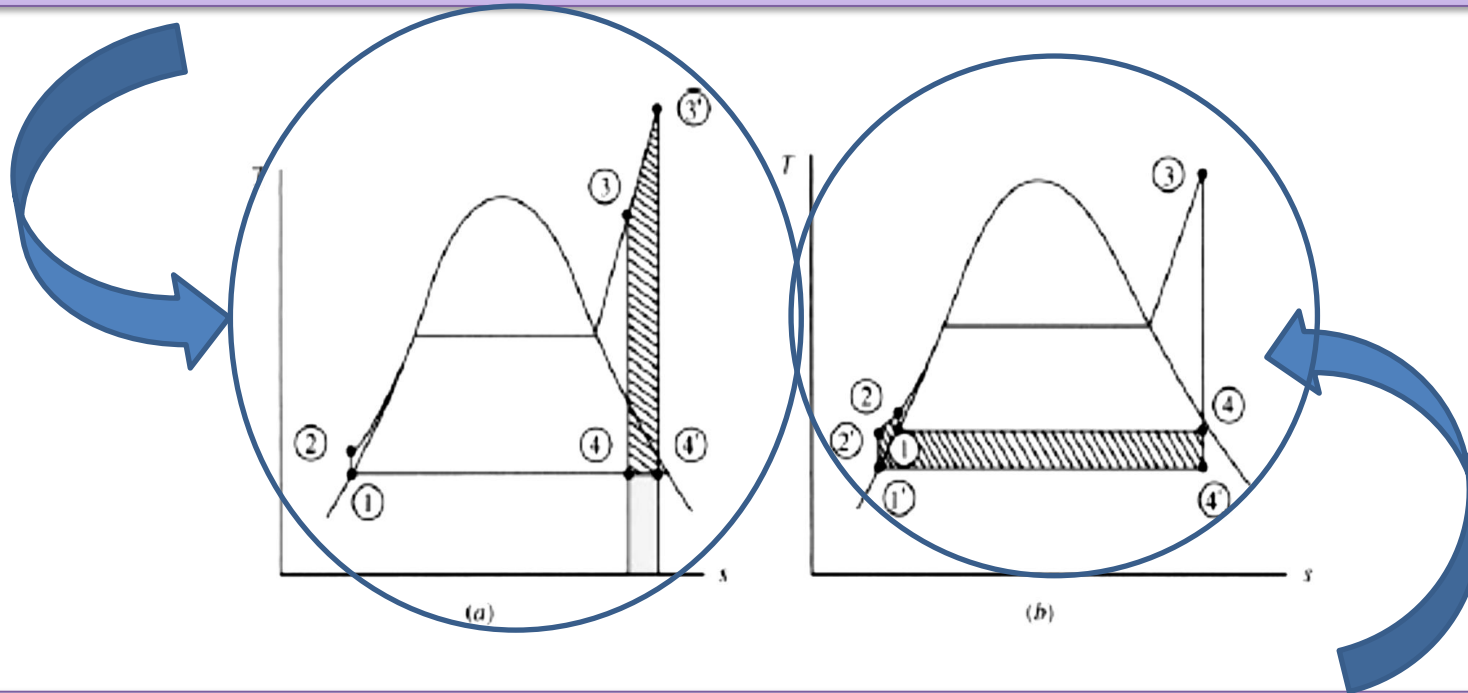
Increasing the boiler pressure while maintaining the **maximum temperature** and the **minimum pressure**.



Problem with moisture at turbine outlet.

# IMPROVING THE RANKINE CYCLE EFFICIENCY

Increasing the maximum temperature – also results in higher steam quality at turbine exit

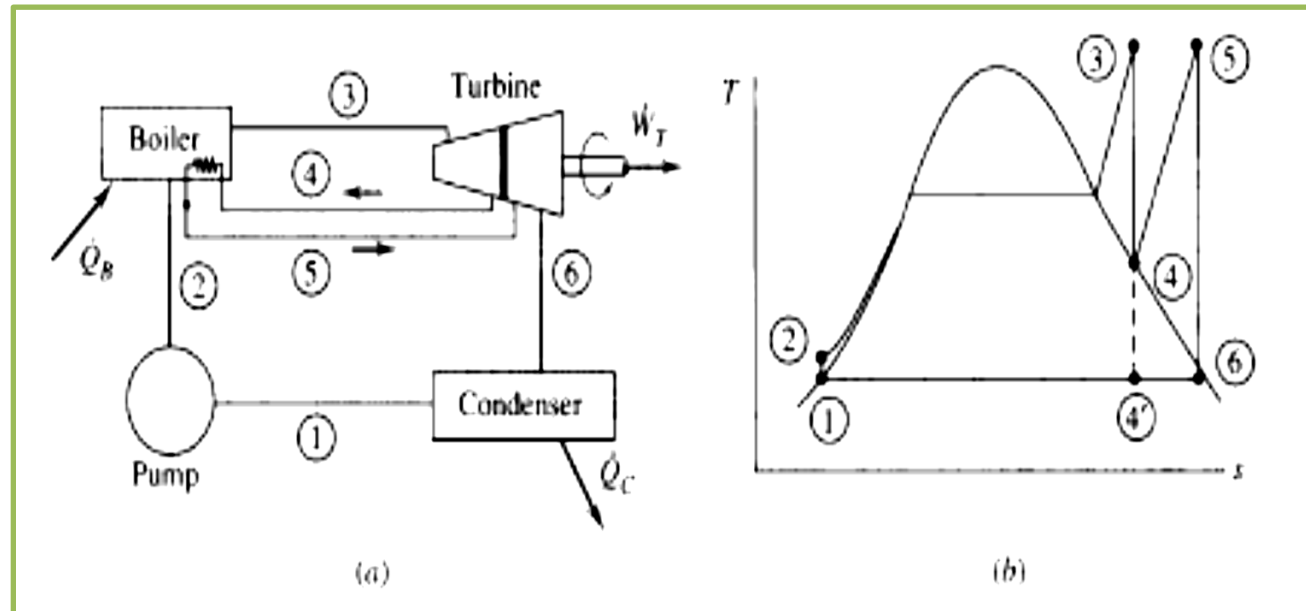


A decrease in condenser pressure improves efficiency, but creates risk of ambient air infiltration into condenser and lower exit steam quality



# IMPROVING THE RANKINE CYCLE EFFICIENCY

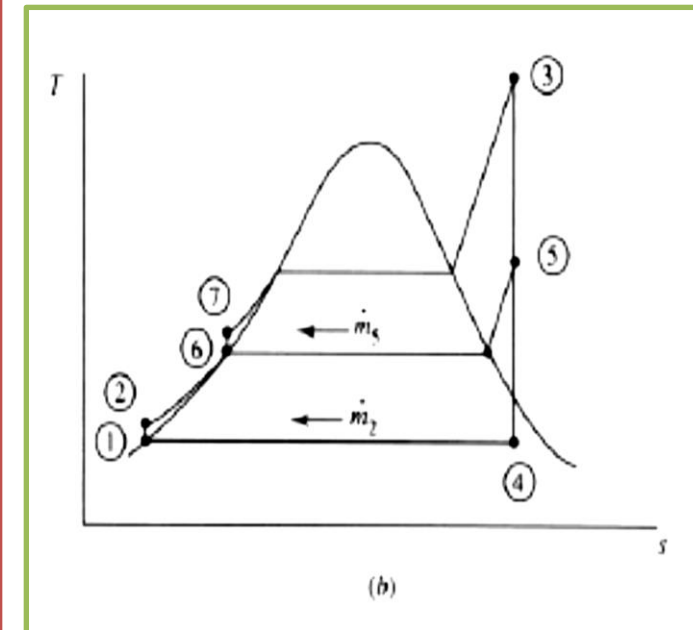
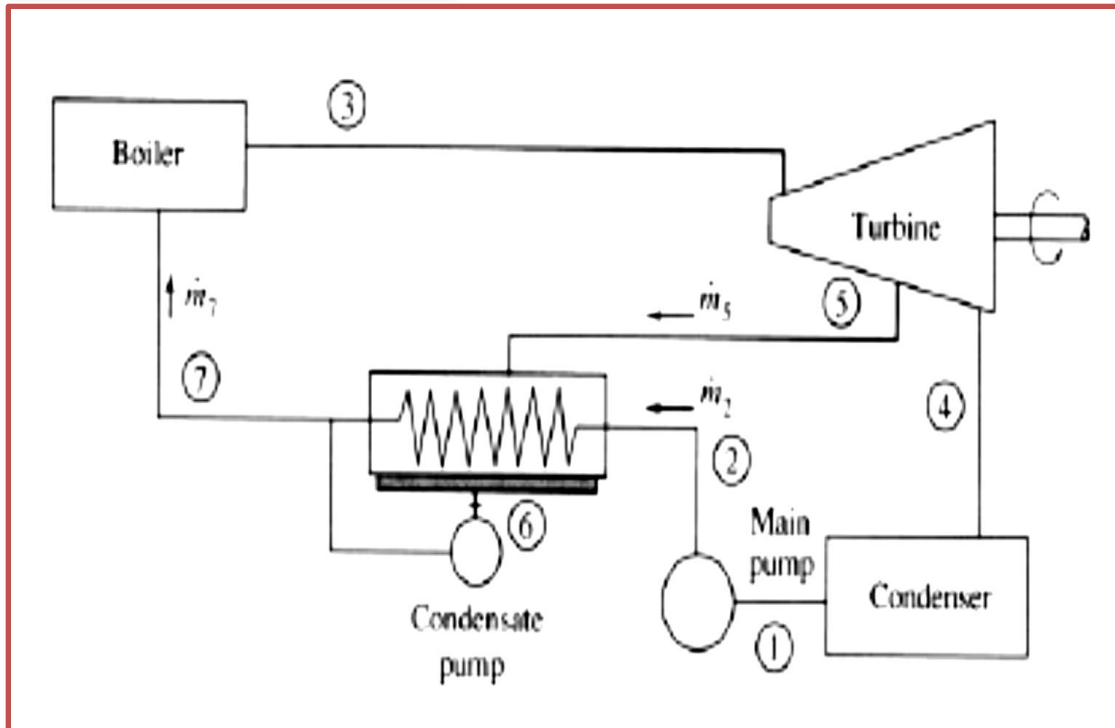
**Reheat** – solves the problem of moisture with increased boiler pressure or reduced condenser pressure.



Rankine cycle with reheat

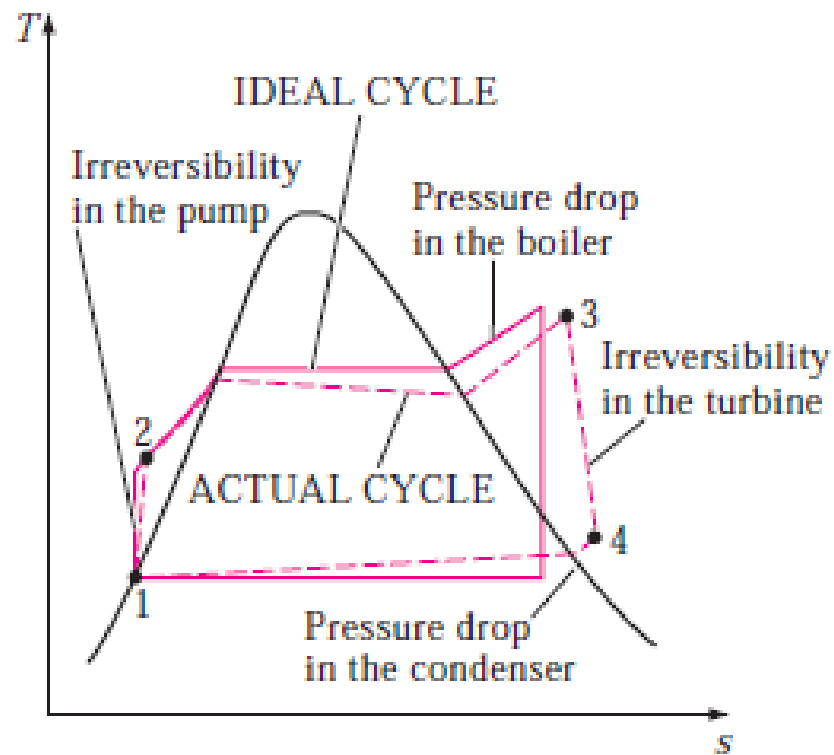
# IMPROVING THE RANKINE CYCLE EFFICIENCY

## Regeneration



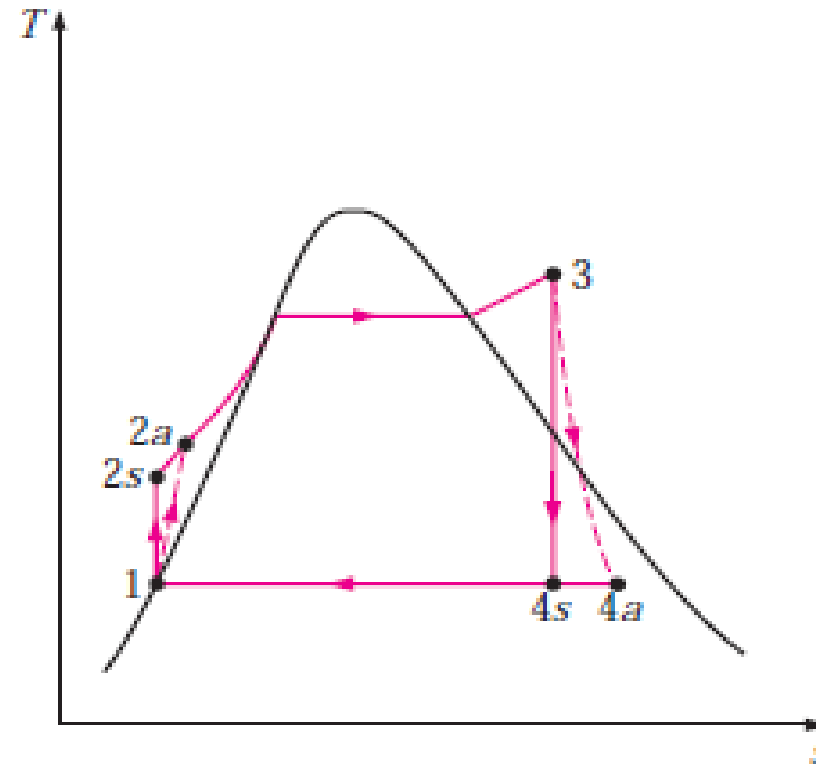
This technique taps some of the steam expanding in the turbine to preheat water entering the boiler. The net effect is a reduced heat input at the boiler side

# EFFECT OF LOSSES ON POWER CYCLE EFFICIENCY



$$\eta_P = \frac{w_s}{w_a} = \frac{h_{2s} - h_1}{h_{2a} - h_1}$$

More input than ideal case.



$$\eta_T = \frac{w_a}{w_s} = \frac{h_3 - h_{4a}}{h_3 - h_{4s}}$$

Less output than ideal case.

