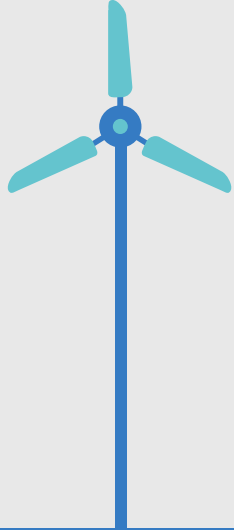




Final Presentation

Krista Roessle, Aleira Sanchez, Max Taubman, Tyler Wisniewski

MAE 4272 Fluids and Heat Transfer Laboratory
Prof. Pepiot
411 - Group 2



Design Objectives

Design Variables

Given

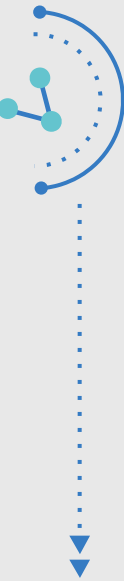
- Radius: $\leq 6''$ from root to tip
- Hub: 1" radius, dovetail shape

Chosen:

- Angular velocity: 1200 rpm
- Far-field wind velocity: 5 m/s
- NACA 4412

Optimization

Maximum power generation
through aerodynamic efficiency



Design Objectives

Design Variables

Given

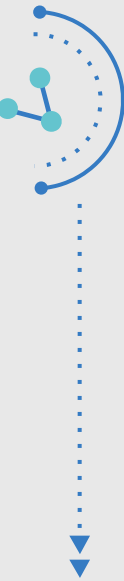
- Radius: $\leq 6"$ from root to tip
- Hub: 1" radius, dovetail shape

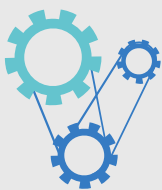
Chosen:

- Angular velocity: 1200 rpm
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- NACA 4412

Optimization

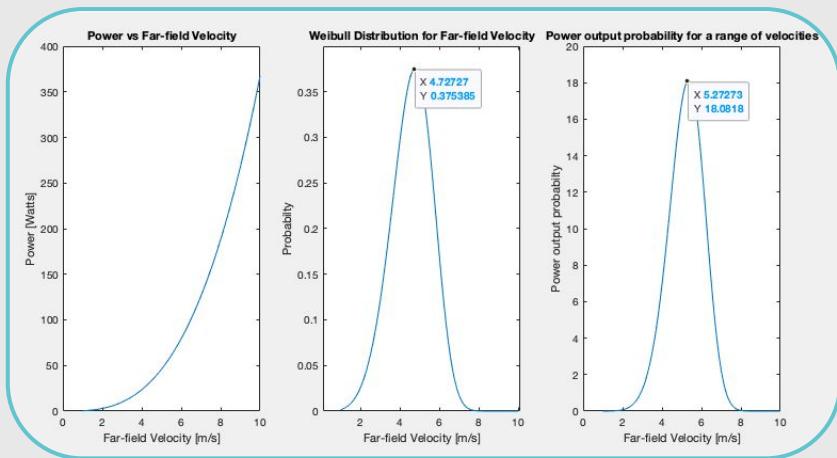
Maximum power generation
through aerodynamic efficiency





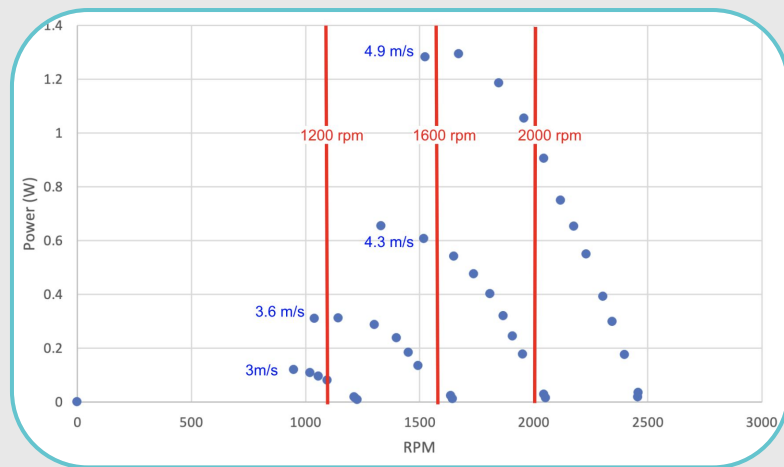
Design Approach

Wind Speed



Speed of most common power output using Weibull distribution

Rotating Rate



Power graph from Lab #6 showing optimal power generation at 1200 rpm





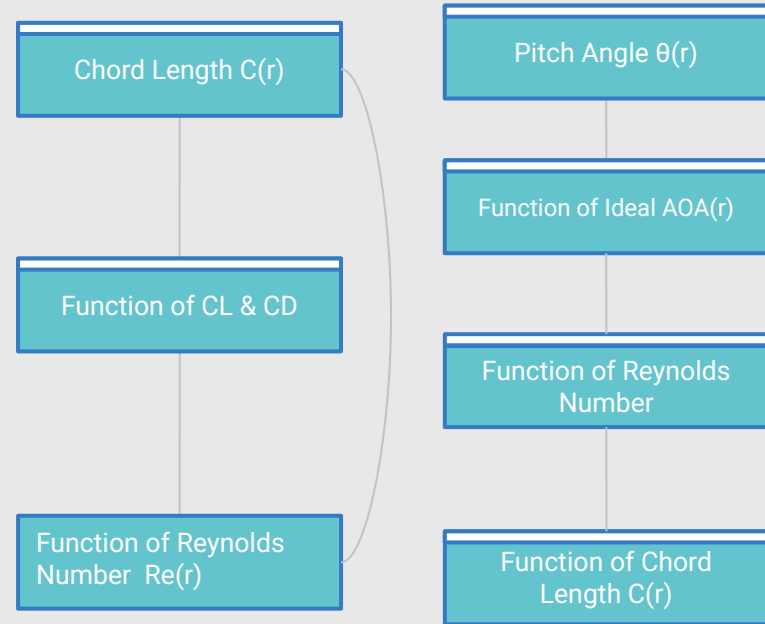
Design Process: The Circular Problem

-In Lab 5, a constant Reynolds number of 100,000 was assumed. This influenced decisions to use constant values for

AOA for our pitch angle calculations
CL & CD for our chord length calculations

-That correspond to maximum CL/CD for the aforementioned Reynolds number.

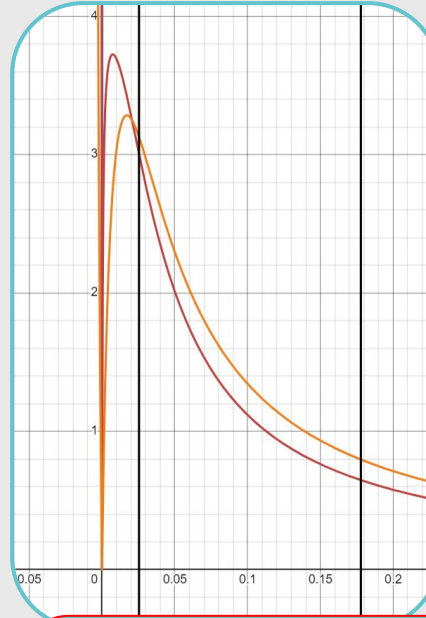
-BUT... Reynolds number is not constant for a given cross section some radial distance r from the hub.



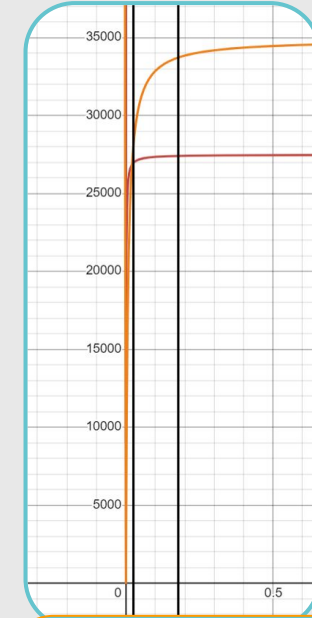


Optimizing Chord Length: An Iterative Process

- Begin by assuming CL and C_D based on the initial Reynolds number 100,000.
- Use these values to find $C(r)_1$ and thus $Re(C(r))_1 = Re(r)_1$. Then took the average of this finding Re_{avg} which was 27300.
- Find the values of CL and C_D corresponding to this Reynolds number and plug them into a new chord length function $C(r)_2$



**2nd Chord Length
(Orange)**
versus
**First Chord Length
(Red)** in inches as a
function of radial
distance in meters



**2nd Reynolds
Number (Orange)**
versus
**1st Reynolds
Number (Red)** as
functions of radial
distance in meters

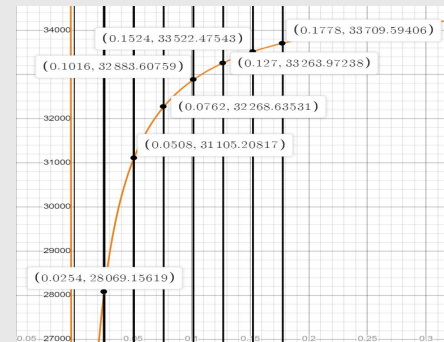




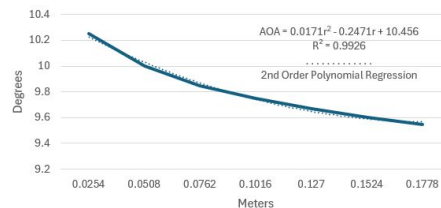
Design Process: Chord Length to Pitch

- Iteratively updated chord length equation $c(r)_2$ factored into our Reynolds number equation $Re(c(r)_2)_2 = Re(r)_2$
- At radial intervals, find the approximate Reynolds numbers and interpolate to find the corresponding angles of attack
- Make a plot for the angle of attacks wrt to r from the hub and perform a regression obtaining angle of attack as a function of r - $AOA(r)$.
- This leads to more accurate pitch angle calculation where not only ϕ but AOA depends on r

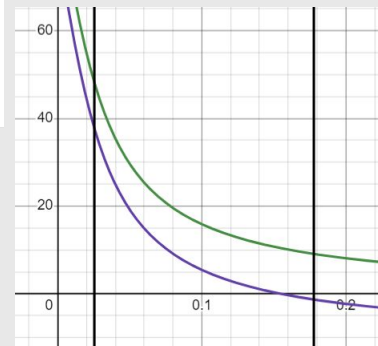
Reynolds Numbers Charted
(iteration 2) at various radial
locations (meters) using $Re(r)_2$



Angle of Attack as a function of Radial Distance
from Hub Center



Angle of attack for Max CL/CD for the
corresponding Reynolds number as a
function of radial distance

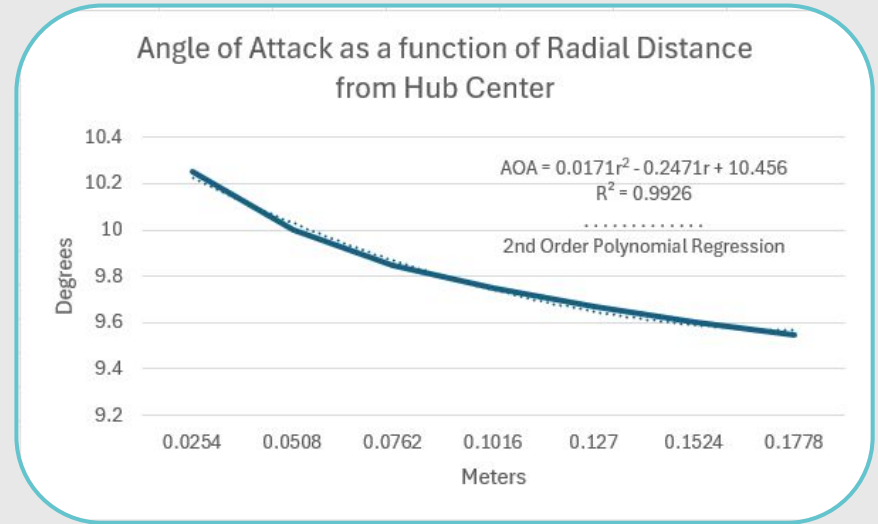




Design Process: Chord Length to Pitch

Incorporating the chord equation:

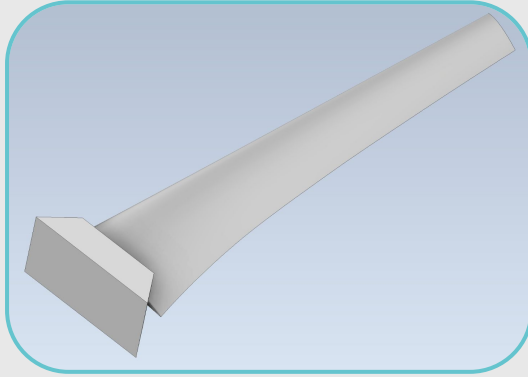
- Found:
 - Angles of attack where C_L/C_D is maximized
 - Angle of attack as a function of radius
- **More accurate pitch angle calculation** that takes into account radial distance and phi



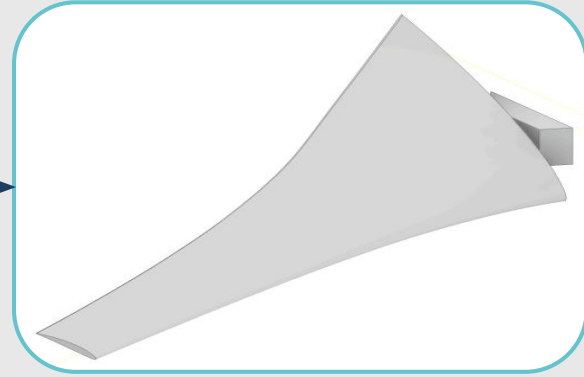


Design Process: CAD

Initial Design



Final Design

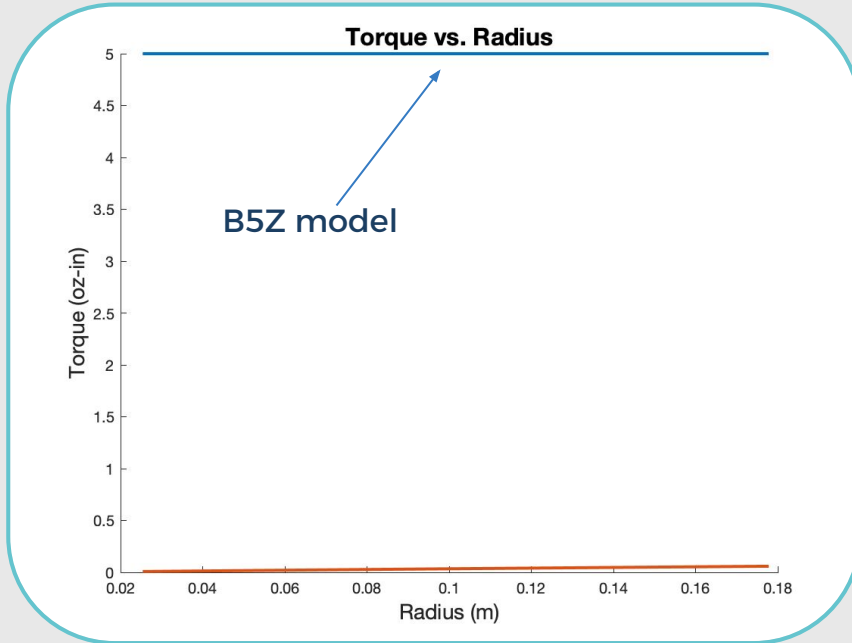


- Utilize Chord Length and Pitch Angle calculations as $f(\text{radius})$
- Establish Work Planes at regular cross sections
- Constrain sketches based on Airfoil Geometry, Pitch Angle, and Chord Length using global variables
 - Enable rapid design iteration
 - Adaptable to changes in circumstances and recovering from errors
- Loft across sketches to create smooth, continuous surface





Constraints



Blade Max Torque: 0.0588 oz-in

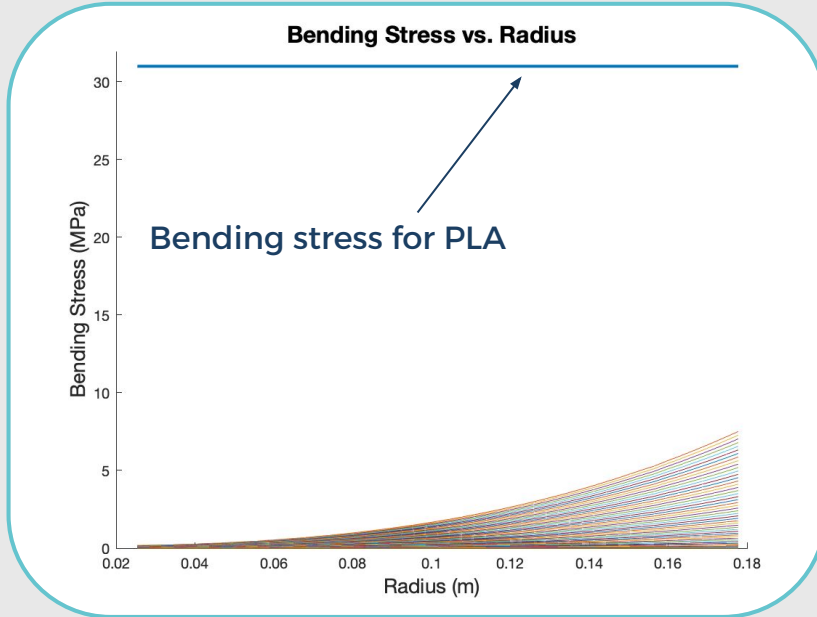
B5Z Max Torque allowed: 5 oz-in

Catastrophic failure? No!





Constraints



Blade Bending Stress: 7.51 MPa

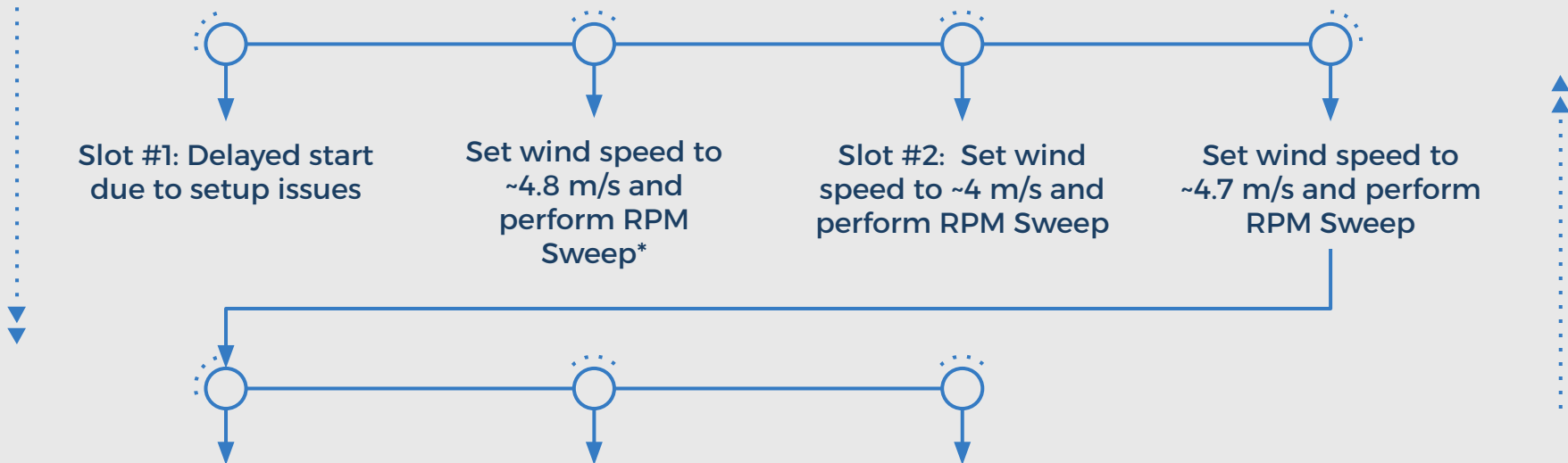
PLA Bending Stress: 31 MPa

Catastrophic failure? No!





Data Collection Process

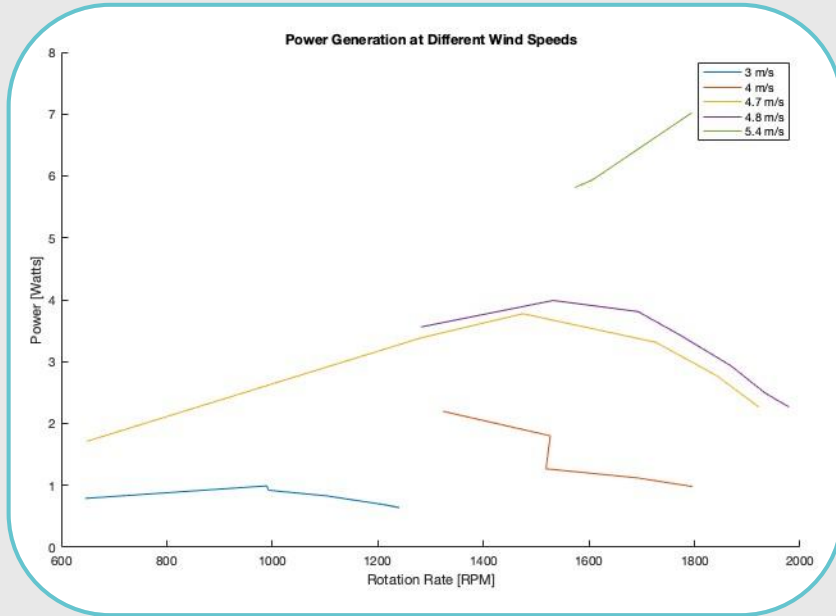


RPM Sweep*: We started at the highest value of rotational rate and incrementally increased torque brake power, collecting data points in between steps. This continued until the rotational rate reached zero.





Actual vs. Theoretical Values for Power



Theoretical

- Max power generation at 4.7 m/s
- Optimal speed for power generation is 1200 RPM

Actual

- Max power generation of **3.98 W** at **4.8 m/s**
- Optimal speed for power generation is around **1570 RPM**
- Due to speed limits with the turbine, 5.4 m/s data set was cut short (greater power gen.)





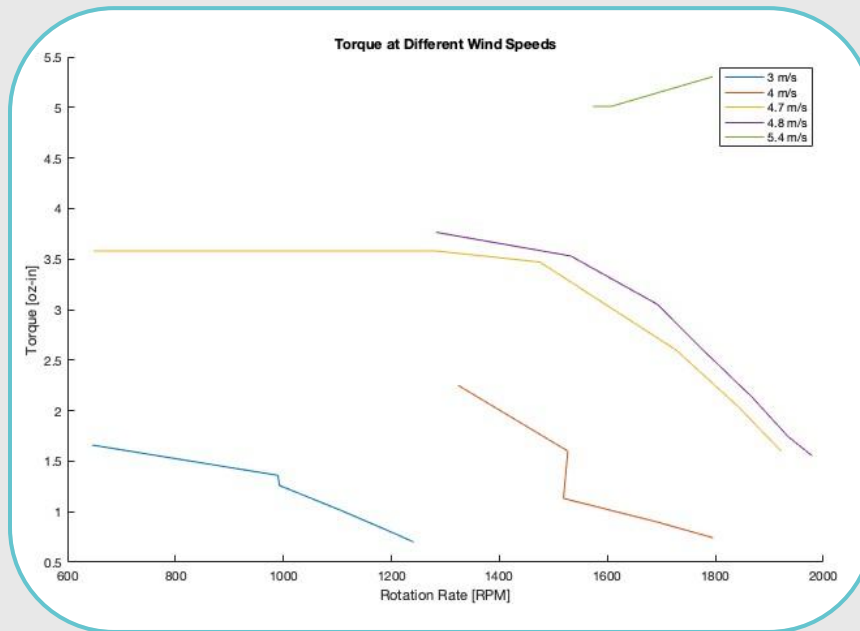
Actual vs Theoretical Values for Torque

Theoretical

- Max torque: 0.0588 oz-in at 5 m/s

Actual

- Max torque: 3.6 oz-in 4.8 m/s
- RPM limited data set for 5.4 m/s
- Discrepancies due to model simplifications in theoretical calculations

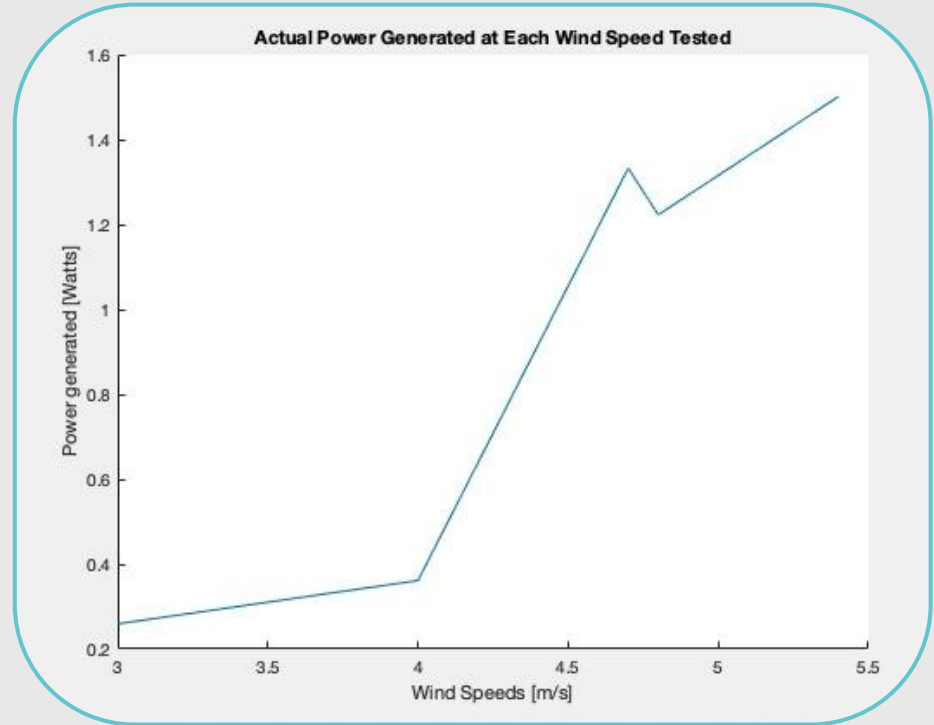




Design Actualization

Operating Rotation Rate

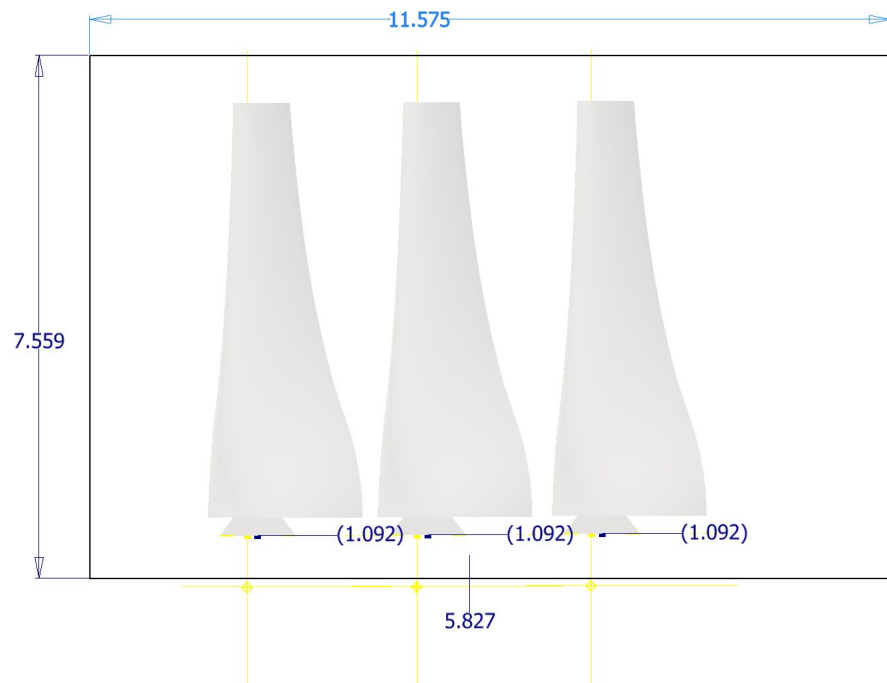
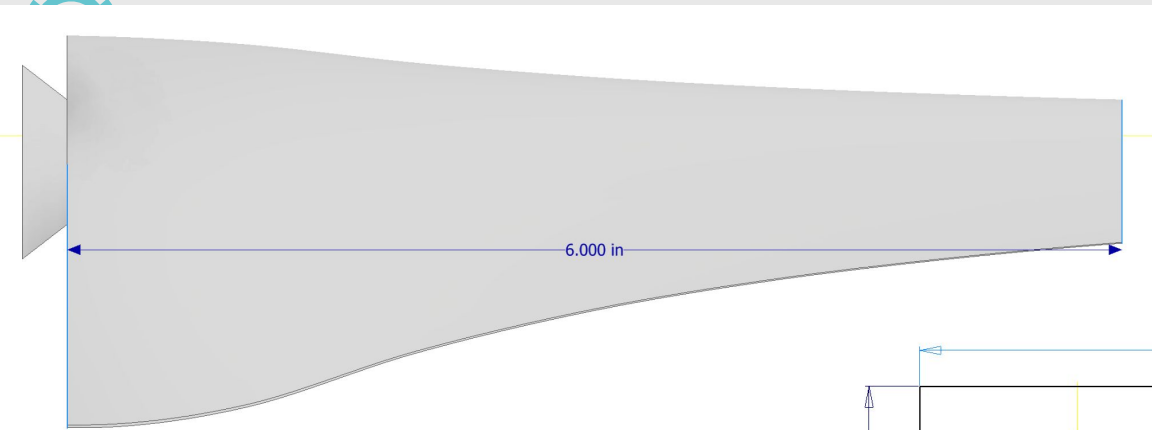
- For each wind speed:
 - Range: 600-1800 RPM
- Wind speeds:
 - 3, 4, 4.7, 4.8 and 5.4 m/s
- Max Power Output
 - ~1.4 W at 4.7 m/s





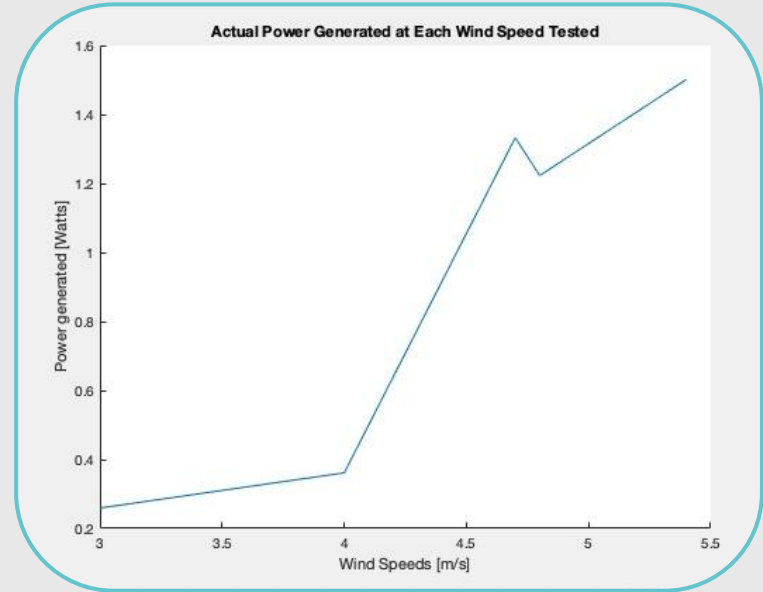
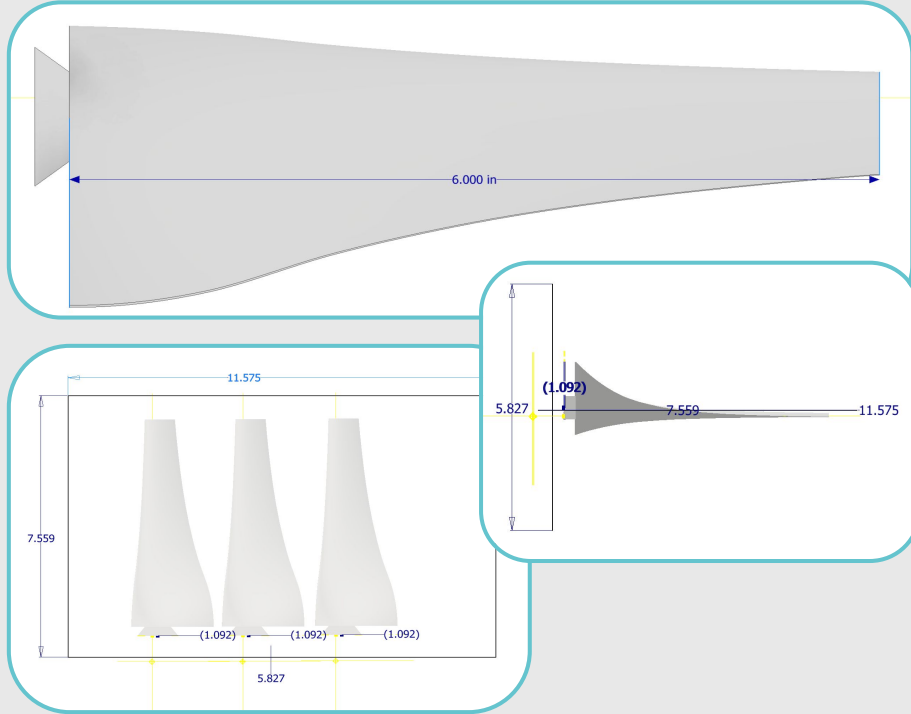
Blade Dimensions







Final Slide: Group 411-2



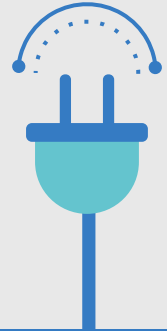
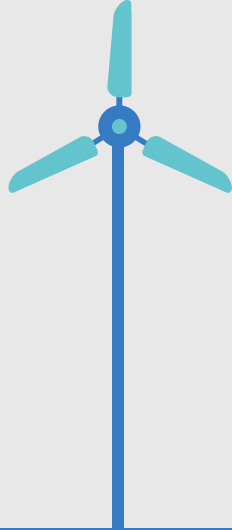
- For each wind speed:
 - Range: 600-1800 RPM
- Wind speeds:
 - 3, 4, 4.7, 4.8 and 5.4 m/s





Thank you for your time!

Questions?



$$R_1 = \frac{(1.18)(L)(c_1)}{1.6(10^{-5})}$$

$$R_2 = \frac{(1.18)(L)(c_2)}{1.6(10^{-5})}$$

$$c_1 = \frac{F^2(4)(a)(1-a)\pi r}{3\left(\frac{1}{2}L_1L^2\cos(\phi) + \frac{1}{2}D_1L^2\sin(\phi)\right)}$$

$$c_2 = \frac{F^2(4)(a)(1-a)\pi r}{3\left(\frac{1}{2}L_2L^2\cos(\phi) + \frac{1}{2}D_2L^2\sin(\phi)\right)}$$

$$F = 5.36$$

$$U = \sqrt{((1-a)F)^2 + (rw(1+A))^2}$$

$$w = 125.663$$

$$A = 0$$

$$\alpha = \frac{1}{3}$$

$$L_1 = 1.14250$$

$$D_1 = 0.01990$$

$$\phi_{\text{phi of pitch angle plus angle of attack}} = \tan^{-1}\left(\frac{(1-a)F}{rw(1+A)}\right)$$

$$\phi = \arctan\left(\frac{(1-a)(F)}{rw(1+A)}\right) - (0.017r^2 - 0.247r + 10.456)$$

$$L_2 = 0.9$$

$$D_2 = 0.195$$

$$x = 1(0.0284)$$

$$x = 7(0.0284)$$

Appendix