3D02_simple_shear_v0p8_A.Flowers_Comments

June 1, 2025

1 Simple shear of a 3D cube

1.0.1 Units

- Length: mm
- Mass: kg
- Time: s
- Force: milliNewtons
- Stress: kPa

1.0.2 Software:

- Dolfinx v0.8.0

In the collection "Example Codes for Coupled Theories in Solid Mechanics,"

By Eric M. Stewart, Shawn A. Chester, and Lallit Anand.

https://solidmechanicscoupledtheories.github.io/

2 Import modules

[1]:
```
# Import FEnicSx/dolfinx
import dolfinx

# For numerical arrays
import numpy as np

# For MPI-based parallelization
from mpi4py import MPI
comm = MPI.COMM_WORLD
rank = comm.Get_rank()

# PETSc solvers
from petsc4py import PETSc

# specific functions from dolfinx modules
from dolfinx import fem, mesh, io, plot, log
```

```
from dolfinx.fem import (Constant, dirichletbc, Function, functionspace,␣
 ↪Expression )
from dolfinx.fem.petsc import NonlinearProblem
from dolfinx.nls.petsc import NewtonSolver
from dolfinx.io import VTXWriter, XDMFFile


# specific functions from ufl modules
import ufl
from ufl import (TestFunctions, TrialFunction, Identity, grad, det, div, dev,␣
 ↪inv, tr, sqrt, conditional ,\
                 gt, dx, inner, derivative, dot, ln, split)

# basix finite elements (necessary for dolfinx v0.8.0)
import basix
from basix.ufl import element, mixed_element, quadrature_element

# Matplotlib for plotting
import matplotlib.pyplot as plt
plt.close('all')

# For timing the code
from datetime import datetime


# Set level of detail for log messages (integer)
# Guide:
# CRITICAL  = 50, // errors that may lead to data corruption
# ERROR     = 40, // things that HAVE gone wrong
# WARNING   = 30, // things that MAY go wrong later
# INFO      = 20, // information of general interest (includes solver info)
# PROGRESS  = 16, // what's happening (broadly)
# TRACE     = 13, // what's happening (in detail)
# DBG       = 10  // sundry
#
log.set_log_level(log.LogLevel.WARNING)
```

3 Define geometry

[2]:
```
# A 3-D cube
length = 1.0 # mm
domain = mesh.create_box(MPI.COMM_WORLD, [[0.0,0.0,0.0],␣
 ↪[length,length,length]],\
                         [8,8,4], mesh.CellType.tetrahedron)

x = ufl.SpatialCoordinate(domain)
```

[ ]:
```
##A.Flowers Comments
length = 1.0 # mm
##Geometry parameter of cube length defined

domain = mesh.create_box(MPI.COMM_WORLD, [[0.0,0.0,0.0],␣
 ↪[length,length,length]],\
##Mesh created of geometry. MPI used as default communicator; includes all␣
 ↪processes used in parallel computation of command.
##Defining the origin of the cube. Lower Corners =0.0; Opposite corners=length␣
 ↪(1.0); Upper corners=1.0

[8,8,4], mesh.CellType.tetrahedron)
##Definition of mesh elements given (i.e. FEA; coarse-vs-fine) and mesh shape
##Number of cells along x,y,z directions. 8 divisions in xand y direction, 4␣
 ↪divisions in z direction

x = ufl.SpatialCoordinate(domain)
##x,y,z spatial coordinates used from boundary conditions
```

Identify boundaries of the domain

[3]:
```
# Identify the planar boundaries of the  box mesh
#
def xBot(x):
    return np.isclose(x[0], 0)
def xTop(x):
    return np.isclose(x[0], length)
def yBot(x):
    return np.isclose(x[1], 0)
def yTop(x):
    return np.isclose(x[1], length)
def zBot(x):
    return np.isclose(x[2], 0)
def zTop(x):
    return np.isclose(x[2], length)

# Mark the sub-domains
boundaries = [(1, xBot),(2,xTop),(3,yBot),(4,yTop),(5,zBot),(6,zTop)]

# build collections of facets on each subdomain and mark them appropriately.
facet_indices, facet_markers = [], [] # initalize empty collections of indices␣
 ↪and markers.
fdim = domain.topology.dim - 1 # geometric dimension of the facet (mesh␣
 ↪dimension - 1)
for (marker, locator) in boundaries:
    facets = mesh.locate_entities(domain, fdim, locator) # an array of all the␣
 ↪facets in a
```

```
                                                    # given subdomain␣
 ↪("locator")
    facet_indices.append(facets)                    # add these facets to␣
 ↪the collection.
    facet_markers.append(np.full_like(facets, marker))   # mark them with the␣
 ↪appropriate index.

# Format the facet indices and markers as required for use in dolfinx.
facet_indices = np.hstack(facet_indices).astype(np.int32)
facet_markers = np.hstack(facet_markers).astype(np.int32)
sorted_facets = np.argsort(facet_indices)
#
# Add these marked facets as "mesh tags" for later use in BCs.
facet_tags = mesh.meshtags(domain, fdim, facet_indices[sorted_facets],␣
 ↪facet_markers[sorted_facets])
```

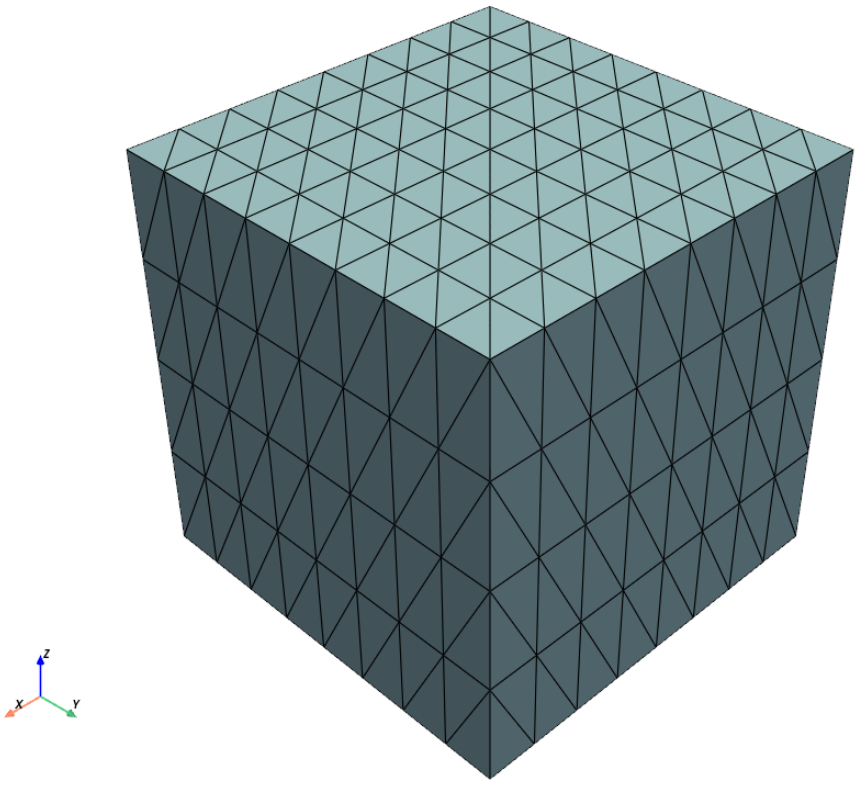
[ ]:
```
##A.Flowers Comments

# Identify the planar boundaries of the  box mesh
def xBot(x):
    return np.isclose(x[0], 0)
##X-coord is close to 0; defines as bottom face in the x-direction
def xTop(x):
    return np.isclose(x[0], length)
##X-coord is close to defined length; defines as top face in x-direction
def yBot(x):
    return np.isclose(x[1], 0)
##Y-coord is close to 0; defines as bottom face in the y-direction
def yTop(x):
    return np.isclose(x[1], length)
##Y-coord is close to defined length; defines as top face in y-direction
def zBot(x):
    return np.isclose(x[2], 0)
##Z-coord is close to 0; defines as bottom face in the z-direction
def zTop(x):
    return np.isclose(x[2], length)
##Z-coord is close to defined length; defines as top face in z-direction
##Defining specific boundary surfaces of a 3D FE model

# Mark the sub-domains
boundaries = [(1, xBot),(2,xTop),(3,yBot),(4,yTop),(5,zBot),(6,zTop)]
##Marks and identifies tag surfaces as an integer of 3D geometry for␣
 ↪application of boundary conditions for FE
##Tags are used to apply Dirichlet / Neumann conidtions

# build collections of facets on each subdomain and mark them appropriately.
```

```
facet_indices, facet_markers = [], [] # initalize empty collections of indices␣
 ↪and markers.
##Initialization; indices stores facets that match specific boundary conditions.
 ↪ Markers stores arrays of same-shape that correspond to specific boundary␣
 ↪marker IDs
fdim = domain.topology.dim - 1 # geometric dimension of the facet (mesh␣
 ↪dimension - 1)
##Calculates dimension of facets with; 3D=dim3, 2D=fdim, with facet edges as 1D␣
 ↪fdim=1
for (marker, locator) in boundaries:
    facets = mesh.locate_entities(domain, fdim, locator) # an array of all the␣
 ↪facets in a
                                                         # given subdomain␣
 ↪("locator")
    facet_indices.append(facets)                         # add these facets to␣
 ↪the collection.
    facet_markers.append(np.full_like(facets, marker))   # mark them with the␣
 ↪appropriate index.
##Loop for defined boundary conditions; locates defined facets of edges /␣
 ↪surfaces of mesh
##Tells solver which boundary conditions to link to each facet. Outcome is␣
 ↪array consisting of indices and markers

# Format the facet indices and markers as required for use in dolfinx.
facet_indices = np.hstack(facet_indices).astype(np.int32)
##Horizontally stacks and flattens all individual arrays into 1D array
##This stores indices in format used for mesh and solver structures
facet_markers = np.hstack(facet_markers).astype(np.int32)
##Stacks / flattens as above
sorted_facets = np.argsort(facet_indices)
##Returns indices that sort indices arrays and reorders the markers to match␣
 ↪appropriate facets
##Prepares data for meshing in next step with appropriate boundary conditions␣
 ↪for specified facets
##Sorting process ensures consistency of facets / markers and helps to avoid␣
 ↪issues in later FE application

# Add these marked facets as "mesh tags" for later use in BCs.
facet_tags = mesh.meshtags(domain, fdim, facet_indices[sorted_facets],␣
 ↪facet_markers[sorted_facets])
##Meshtag is data structure associating build of mech components (i.e. facets␣
 ↪and integers). These markers are used to apply specified boundary conditions
##This allows for fixed boundary conditions and force / displacement to be␣
 ↪applied
```

Visualize reference configuration and boundary facets

[4]:
```
import pyvista
pyvista.set_jupyter_backend('html')
from dolfinx.plot import vtk_mesh
pyvista.start_xvfb()

# initialize a plotter
plotter = pyvista.Plotter()

# Add the mesh
topology, cell_types, geometry = plot.vtk_mesh(domain, domain.topology.dim)
grid = pyvista.UnstructuredGrid(topology, cell_types, geometry)
plotter.add_mesh(grid, show_edges=True)

labels = dict(zlabel='Z', xlabel='X', ylabel='Y')
plotter.add_axes(**labels)

plotter.screenshot("mesh.png")

from IPython.display import Image
Image(filename='mesh.png')
```

[4]:



[ ]:
```
##A.Flowers Comments

import pyvista
pyvista.set_jupyter_backend('html')
from dolfinx.plot import vtk_mesh
pyvista.start_xvfb()
##Jupyter notebook display of FE meshes (non-GUI format)

# initialize a plotter
plotter = pyvista.Plotter()
##Creates 3D renderings

# Add the mesh
topology, cell_types, geometry = plot.vtk_mesh(domain, domain.topology.dim)
##Conversion of Dolfinx mesh topology (element nodes), cell types (element␣
 ↪type), and geometry (point coordinates) to PyVista format
grid = pyvista.UnstructuredGrid(topology, cell_types, geometry)
##PyVista format that renders grid for unstructured meshes (i.e. FEM)
plotter.add_mesh(grid, show_edges=True)
##Adds mesh for visualization

labels = dict(zlabel='Z', xlabel='X', ylabel='Y')
plotter.add_axes(**labels)
##Labeling of plot

plotter.screenshot("mesh.png")
##Saves images to path

from IPython.display import Image
Image(filename='mesh.png')
##Displays images within Jupyter notebook
```

3.1 Define boundary and volume integration measure

[5]:
```
# Surface labels from gmsh:
# Physical Surface("xbot", 33)
# Physical Surface("ybot", 34)
# Physical Surface("xtop", 35)

# Define the boundary integration measure "ds" using the facet tags,
# also specify the number of surface quadrature points.
ds = ufl.Measure('ds', domain=domain, subdomain_data=facet_tags,␣
 ↪metadata={'quadrature_degree':4})

# Define the volume integration measure "dx"
# also specify the number of volume quadrature points.
dx = ufl.Measure('dx', domain=domain, metadata={'quadrature_degree': 4})
```

```
#  Define facet normal
n = ufl.FacetNormal(domain)
```

[ ]:
```
##A.Flowers Comments

# Surface labels from gmsh:
# Physical Surface("xbot", 33)
# Physical Surface("ybot", 34)
# Physical Surface("xtop", 35)

# Define the boundary integration measure "ds" using the facet tags,
# also specify the number of surface quadrature points.
ds = ufl.Measure('ds', domain=domain, subdomain_data=facet_tags,␣
 ↪metadata={'quadrature_degree':4})
##UFL used in DolfinX and FEniCS to define symbolic integral measures; tells␣
 ↪solver how and where to integrate expressions in weak forms
##ds= defining boundary measure integration over facets
##quadrature= accuracy of numerical integration; high degree= more accurate and␣
 ↪more computation

# Define the volume integration measure "dx"
# also specify the number of volume quadrature points.
dx = ufl.Measure('dx', domain=domain, metadata={'quadrature_degree': 4})
##dx=volume domain

#  Define facet normal
n = ufl.FacetNormal(domain)
##Expression for outward unit normal vector on boundary facets; vector shape is␣
 ↪same as mesh dimension. Used for Neumann boundary conditions
```

4 Material parameters

-Arruda-Boyce model

[6]:
```
Gshear_0 = Constant(domain,PETSc.ScalarType(280.0))             # Ground state␣
 ↪shear modulus
lambdaL  = Constant(domain,PETSc.ScalarType(5.12))              # Locking stretch
Kbulk    = Constant(domain,PETSc.ScalarType(1000.0*Gshear_0))
```

5 Simulation time-control related params

[7]:
```
# Cyclical displacement history parameters
gammaAmp = 1.0                 # amplitude of shear strain
uMax     =  length * gammaAmp # amplitude of displacement. Remember L is the␣
 ↪box size
```

```
#
ttd      = 2.5                # quarter-cycle time for the sinusoidal input
T_cycle  = 4.0*ttd            # cycle time
omega    = 2.* np.pi /T_cycle # frequency in radians per sec
n_cycles = 2                  # Number of cycles

# Total time
Ttot = n_cycles*T_cycle

# start time at t=0
t = 0

# Time step
dt = 0.2

# Subroutine for displacing the top surface:
def dispRamp(t):
    return uMax * np.sin(omega*t)
```

6 Function spaces

[8]:
```
# Define function space, both vectorial and scalar
U2 = element("Lagrange", domain.basix_cell(), 2, shape=(3,)) # For displacement
P1 = element("Lagrange", domain.basix_cell(), 1)             # For  pressure
#
TH = mixed_element([U2, P1])      # Taylor-Hood style mixed element
ME = functionspace(domain, TH)    # Total space for all DOFs

# Define actual functions with the required DOFs
w    = Function(ME)
u, p = split(w)  # displacement u, pressure p

# A copy of functions to store values in the previous step
w_old         = Function(ME)
u_old,  p_old = split(w_old)

# Define test functions
u_test, p_test = TestFunctions(ME)

# Define trial functions needed for automatic differentiation
dw = TrialFunction(ME)
```

7 Initial conditions

- The initial conditions for degrees of freedom u and p are zero everywhere

- These are imposed automatically, since we have not specified any non-zero initial conditions.

8 Subroutines for kinematics and constitutive equations

[10]:
```
# Deformation gradient
def F_calc(u):
    Id = Identity(3)
    F  = Id + grad(u)
    return F

def lambdaBar_calc(u):
    F = F_calc(u)
    C = F.T*F
    Cdis = J**(-2/3)*C
    I1 = tr(Cdis)
    lambdaBar = sqrt(I1/3.0)
    return lambdaBar

def zeta_calc(u):
    lambdaBar = lambdaBar_calc(u)
    # Use Pade approximation of Langevin inverse
    z    = lambdaBar/lambdaL
    z    = conditional(gt(z,0.95), 0.95, z) # Keep simulation from blowing up
    beta = z*(3.0 - z**2.0)/(1.0 - z**2.0)
    zeta = (lambdaL/(3*lambdaBar))*beta
    return zeta

# Generalized shear modulus for Arruda-Boyce model
def Gshear_AB_calc(u):
    zeta    = zeta_calc(u)
    Gshear  = Gshear_0 * zeta
    return Gshear

#-------------------------------------------
# Subroutine for calculating the Cauchy stress
#-------------------------------------------
def T_calc(u,p):
    Id = Identity(3)
    F   = F_calc(u)
    J = det(F)
    B = F*F.T
    Bdis = J**(-2/3)*B
    Gshear  = Gshear_AB_calc(u)
    T = (1/J)* Gshear * dev(Bdis) - p * Id
    return T

#-------------------------------------------
```

```
# Subroutine for calculating the Piola  stress
#-------------------------------------------
def Piola_calc(u, p):
    Id = Identity(3)
    F   = F_calc(u)
    J = det(F)
    #
    T   = T_calc(u,p)
    #
    Tmat   = J * T * inv(F.T)
    return Tmat
```

[ ]:
```
##A.Flowers Comments

# Deformation gradient
def F_calc(u):
    Id = Identity(3)
    F  = Id + grad(u)
    return F
##Calculation for deformation gradient tensor F

def lambdaBar_calc(u):
    F = F_calc(u)
    C = F.T*F
    Cdis = J**(-2/3)*C
    I1 = tr(Cdis)
    lambdaBar = sqrt(I1/3.0)
    return lambdaBar
##Scalar stretch measure used in hyperelasticity models

def zeta_calc(u):
    lambdaBar = lambdaBar_calc(u)
##Isochoric stretch from deformation
    # Use Pade approximation of Langevin inverse
    z    = lambdaBar/lambdaL
##Normalizes stretch from polymer network
    z    = conditional(gt(z,0.95), 0.95, z) # Keep simulation from blowing up
##Prevents numeric instability; Langevin function because singular
    beta = z*(3.0 - z**2.0)/(1.0 - z**2.0)
##Pade approximation; used for stability
    zeta = (lambdaL/(3*lambdaBar))*beta
    return zeta
##Stress scalar from statistical mechanics model for polymers; accounts for␣
 ↪finite chain extensibility. Stress tensors for nonlinear chain elasticity

# Generalized shear modulus for Arruda-Boyce model
def Gshear_AB_calc(u):
```

```
##Effective shear for nonlinear hyperelastic material
    zeta    = zeta_calc(u)
##Stretch dependent factor using inverse Langevin. Increasing of stretch=␣
 ↪polymers stiffen
    Gshear  = Gshear_0 * zeta
    return Gshear
##Shear module grows due to deformation
##This is important due to modeling with biological materials (i.e. tissue);␣
 ↪nonlinear and stretch sensitive

#-----------------------------------------------
# Subroutine for calculating the Cauchy stress
#-----------------------------------------------
def T_calc(u,p):
    Id = Identity(3)
    F   = F_calc(u)
##Deformation gradient
    J = det(F)
##Jacobian (volume change due to deformation)
    B = F*F.T
##Cauchy-Green tensor
    Bdis = J**(-2/3)*B
##Removes volumetric part to get isochoric (volume perserving aspect)
    Gshear  = Gshear_AB_calc(u)
##Stretch dependent shear
    T = (1/J)* Gshear * dev(Bdis) - p * Id
    return T
##Cauchy stress calculation; shape change and pressure separated to obtain␣
 ↪deformed configuration

#-----------------------------------------------
# Subroutine for calculating the Piola  stress
#-----------------------------------------------
def Piola_calc(u, p):
    Id = Identity(3)
    F   = F_calc(u)
    J = det(F)
    T   = T_calc(u,p)
    Tmat   = J * T * inv(F.T)
    return Tmat
##Piola stress used in weak form of balance equation, with displacement␣
 ↪gradient; defined in terms of reference coordinates
```

9 Evaluate kinematics and constitutive relations

[11]:
```
F =  F_calc(u)
J = det(F)
lambdaBar = lambdaBar_calc(u)

# Piola stress
Tmat = Piola_calc(u, p)
```

[ ]:
```
##A.Flowers Comments

F =  F_calc(u)
##F= deformation gradient tensor; u= displacement (unkown and solving for)
J = det(F)
##J= Jacobian determinant. Volume change during deformation
lambdaBar = lambdaBar_calc(u)
##incompressible hyperelasticity; seperates volumetric deviatoric, shape␣
 ↪changing parts of deformation. Volume corrected stretch is calculated for␣
 ↪use in isochoric strain energy

# Piola stress
Tmat = Piola_calc(u, p)
##Computes Piola stress tensor from displacement field (u) and pressure (p)
```

10 Weak forms

[12]:
```
# Residuals:
# Res_0: Balance of forces (test fxn: u)
# Res_1: Coupling pressure (test fxn: p)

# The weak form for the equilibrium equation. No body force
Res_0 = inner(Tmat , grad(u_test) )*dx

# The weak form for the pressure
fac_p = ln(J)/J
#
Res_1 = dot( (p/Kbulk + fac_p), p_test)*dx

# Total weak form
Res = Res_0 +  Res_1

# Automatic differentiation tangent:
a = derivative(Res, w, dw)
```

[ ]:
```
##A.Flowers Comments

# Residuals:
```

```
# Res_0: Balance of forces (test fxn: u)
# Res_1: Coupling pressure (test fxn: p)

# The weak form for the equilibrium equation. No body force
Res_0 = inner(Tmat , grad(u_test) )*dx
##Mechanical residual of weak form for nonlinear elasticity; used to build the␣
 ↪residual vector. Used in FE for a deforming solid

# The weak form for the pressure
fac_p = ln(J)/J
##Scalar factor used for compressible / incompressible materials due to␣
 ↪pressure / energy in nonlinear elasticity

Res_1 = dot( (p/Kbulk + fac_p), p_test)*dx
##Residual defined for pressure field in FE due to incompressible materials.␣
 ↪Differentiates volumetric strain energy

# Total weak form
Res = Res_0 +  Res_1
##Defines total residual of weak form; from force balance (linear momentum /␣
 ↪mechanical equillibrium) and incompressibility (pressure equation)

# Automatic differentiation tangent:
a = derivative(Res, w, dw)
##Jacobian form to solve for nonlinear PDE
```

11 Set-up output files

[13]:
```
# results file name
results_name = "3D_simple_shear"

# Function space for projection of results
U1 = element("DG", domain.basix_cell(), 1, shape=(3,)) # For displacement
P0 = element("DG", domain.basix_cell(), 1)             # For  pressure

V2 = fem.functionspace(domain, U1) #Vector function space
V1 = fem.functionspace(domain, P0) #Scalar function space

# fields to write to output file
u_vis = Function(V2)
u_vis.name = "disp"

p_vis = Function(V1)
p_vis.name = "p"

J_vis = Function(V1)
```

```
J_vis.name = "J"
J_expr = Expression(J,V1.element.interpolation_points())

lambdaBar_vis = Function(V1)
lambdaBar_vis.name = "lambdaBar"
lambdaBar_expr = Expression(lambdaBar,V1.element.interpolation_points())

P11 = Function(V1)
P11.name = "P11"
P11_expr = Expression(Tmat[0,0],V1.element.interpolation_points())
P22 = Function(V1)
P22.name = "P22"
P22_expr = Expression(Tmat[1,1],V1.element.interpolation_points())
P33 = Function(V1)
P33.name = "P33"
P33_expr = Expression(Tmat[2,2],V1.element.interpolation_points())
P21 = Function(V1)
P21.name = "P21"
P21_expr = Expression(Tmat[1,0],V1.element.interpolation_points())

T   = Tmat*F.T/J
T0   = T - (1/3)*tr(T)*Identity(3)
Mises = sqrt((3/2)*inner(T0, T0))
Mises_vis= Function(V1,name="Mises")
Mises_expr = Expression(Mises,V1.element.interpolation_points())

# set up the output VTX files.
file_results = VTXWriter(
    MPI.COMM_WORLD,
    "results/" + results_name + ".bp",
    [  # put the functions here you wish to write to output
        u_vis, p_vis, J_vis, P11, P22, P33, P21,
        lambdaBar_vis, Mises_vis,
    ],
    engine="BP4",
)

def writeResults(t):
       # Output field interpolation
       u_vis.interpolate(w.sub(0))
       p_vis.interpolate(w.sub(1))
       J_vis.interpolate(J_expr)
       P11.interpolate(P11_expr)
       P22.interpolate(P22_expr)
       P33.interpolate(P33_expr)
       P21.interpolate(P21_expr)
       lambdaBar_vis.interpolate(lambdaBar_expr)
```

```
        Mises_vis.interpolate(Mises_expr)

        # Write output fields
        file_results.write(t)
```

12 Infrastructure for pulling out time history data (force, displacement, etc.)

[14]:
```
# v0.8.0 syntax:
pointForDisp = np.array([length,length,length])


bb_tree = dolfinx.geometry.bb_tree(domain,domain.topology.dim)
cell_candidates = dolfinx.geometry.compute_collisions_points(bb_tree,␣
 ↪pointForDisp)

# v0.8.0 syntax:
colliding_cells = dolfinx.geometry.compute_colliding_cells(domain,␣
 ↪cell_candidates, pointForDisp).array

# Computing the shear reaction force
traction      = dot(Tmat,n)
tangent       = ufl.as_vector([1,0,0])
shearRxnForce = fem.form(dot(traction,tangent)*ds(4))

# Recall the boundary definitions:
# boundaries = [(1, xBot),(2,xTop),(3,yBot),(4,yTop),(5,zBot),(6,zTop)]
```

13 Name the analysis step

[15]:
```
# Give the step a descriptive name
step = "Shear"
```

13.1 Boundary condtions

[16]:
```
# # Recall the boundary definitions:
# # boundaries = [(1, xBot),(2,xTop),(3,yBot),(4,yTop),(5,zBot),(6,zTop)]

# Constant for applied displacement
disp_cons = Constant(domain,PETSc.ScalarType(dispRamp(0)))

# Find the specific DOFs which will be constrained.
yBot_u1_dofs = fem.locate_dofs_topological(ME.sub(0).sub(0), facet_tags.dim,␣
 ↪facet_tags.find(3))
```

```
yBot_u2_dofs = fem.locate_dofs_topological(ME.sub(0).sub(1), facet_tags.dim,␣
 ↪facet_tags.find(3))
yBot_u3_dofs = fem.locate_dofs_topological(ME.sub(0).sub(2), facet_tags.dim,␣
 ↪facet_tags.find(3))
#
yTop_u1_dofs = fem.locate_dofs_topological(ME.sub(0).sub(0), facet_tags.dim,␣
 ↪facet_tags.find(4))
yTop_u2_dofs = fem.locate_dofs_topological(ME.sub(0).sub(1), facet_tags.dim,␣
 ↪facet_tags.find(4))
yTop_u3_dofs = fem.locate_dofs_topological(ME.sub(0).sub(2), facet_tags.dim,␣
 ↪facet_tags.find(4))

# building Dirichlet BCs
bcs_1 = dirichletbc(0.0, yBot_u1_dofs, ME.sub(0).sub(0))  # u1 fix - yBot
bcs_2 = dirichletbc(0.0, yBot_u2_dofs, ME.sub(0).sub(1))  # u2 fix - yBot
bcs_3 = dirichletbc(0.0, yBot_u3_dofs, ME.sub(0).sub(2))  # u3 fix - yBot
#
bcs_4 = dirichletbc(disp_cons, yTop_u1_dofs, ME.sub(0).sub(0))  # u1 disp ramp␣
 ↪- yTop
bcs_5 = dirichletbc(0.0, yTop_u2_dofs, ME.sub(0).sub(1))  # u2 fix - yTop
bcs_6 = dirichletbc(0.0, yTop_u3_dofs, ME.sub(0).sub(2))  # u2 fix - yTop

bcs = [bcs_1, bcs_2, bcs_3, bcs_4, bcs_5, bcs_6]
```

[ ]:
```
##A.Flowers

# # Recall the boundary definitions:
# # boundaries = [(1, xBot),(2,xTop),(3,yBot),(4,yTop),(5,zBot),(6,zTop)]

# Constant for applied displacement
disp_cons = Constant(domain,PETSc.ScalarType(dispRamp(0)))
##Gives constant displacement value; Application of Dirichlet to FE
##Uniform scalar applied to mesh domain

# Find the specific DOFs which will be constrained.
yBot_u1_dofs = fem.locate_dofs_topological(ME.sub(0).sub(0), facet_tags.dim,␣
 ↪facet_tags.find(3))
yBot_u2_dofs = fem.locate_dofs_topological(ME.sub(0).sub(1), facet_tags.dim,␣
 ↪facet_tags.find(3))
yBot_u3_dofs = fem.locate_dofs_topological(ME.sub(0).sub(2), facet_tags.dim,␣
 ↪facet_tags.find(3))
##DoFs defined for boundary surface of mesh due to displacement field in each␣
 ↪direction on bottom surface of the cube
##Fixes the base the of cube
```

```
yTop_u1_dofs = fem.locate_dofs_topological(ME.sub(0).sub(0), facet_tags.dim,␣
 ↪facet_tags.find(4))
yTop_u2_dofs = fem.locate_dofs_topological(ME.sub(0).sub(1), facet_tags.dim,␣
 ↪facet_tags.find(4))
yTop_u3_dofs = fem.locate_dofs_topological(ME.sub(0).sub(2), facet_tags.dim,␣
 ↪facet_tags.find(4))
##DoFs defined for boundary surface of mesh due to displacement field in each␣
 ↪direction on top surface of the cube
##Applies displcement load on top surface of the cube (i.e. shear deformation)

# building Dirichlet BCs
bcs_1 = dirichletbc(0.0, yBot_u1_dofs, ME.sub(0).sub(0))  # u1 fix - yBot
bcs_2 = dirichletbc(0.0, yBot_u2_dofs, ME.sub(0).sub(1))  # u2 fix - yBot
bcs_3 = dirichletbc(0.0, yBot_u3_dofs, ME.sub(0).sub(2))  # u3 fix - yBot
##Displacement field constraints set to all parts y=0 (bottom surface); fixing␣
 ↪to 0 ensures full clamped boundary condition on bottom surface of the cube

bcs_4 = dirichletbc(disp_cons, yTop_u1_dofs, ME.sub(0).sub(0))  # u1 disp ramp␣
 ↪- yTop
##Applies  horizontal displacement in the x-direction on the top surface of the␣
 ↪cube
##Key driving condition of shear deformation
bcs_5 = dirichletbc(0.0, yTop_u2_dofs, ME.sub(0).sub(1))  # u2 fix - yTop
##Sets y-displacement on top surface; Ensures pure shear as to not obtain any␣
 ↪vertical strain by keeping vertical height set
bcs_6 = dirichletbc(0.0, yTop_u3_dofs, ME.sub(0).sub(2))  # u2 fix - yTop
##Sets z-displacement on top surface; Ensures surface remains in-plane and␣
 ↪prevents warping

bcs = [bcs_1, bcs_2, bcs_3, bcs_4, bcs_5, bcs_6]
## Applies boundary conditions to the nonlinear solver
```

13.2 Define the nonlinear variational problem

[17]:
```
# # Optimization options for the form compiler

# Set up nonlinear problem
problem = NonlinearProblem(Res, w, bcs, a)

# the global newton solver and params
solver = NewtonSolver(MPI.COMM_WORLD, problem)
solver.convergence_criterion = "incremental"
solver.rtol = 1e-8
solver.atol = 1e-8
solver.max_it = 50
solver.report = True
```

```
#  The Krylov solver parameters.
ksp = solver.krylov_solver
opts = PETSc.Options()
option_prefix = ksp.getOptionsPrefix()
opts[f"{option_prefix}ksp_type"] = "preonly" # "preonly" works equally well
opts[f"{option_prefix}pc_type"] = "lu" # do not use 'gamg' pre-conditioner
opts[f"{option_prefix}pc_factor_mat_solver_type"] = "mumps"
opts[f"{option_prefix}ksp_max_it"] = 30
ksp.setFromOptions()
```

13.3 Start calculation loop

[18]:
```
# Variables for storing time history
totSteps = 100000
timeHist0 = np.zeros(shape=[totSteps])
timeHist1 = np.zeros(shape=[totSteps])
timeHist2 = np.zeros(shape=[totSteps])

#Iinitialize a counter for reporting data
ii=0

# Write initial state to file
writeResults(t=0.0)

# Print out message for simulation start
print("-------------------------------------")
print("Simulation Start")
print("-------------------------------------")
# Store start time
startTime = datetime.now()

# Time-stepping solution procedure loop
while (round(t + dt, 9) <= Ttot):

    # increment time
    t += dt
    # increment counter
    ii += 1

    # update time variables in time-dependent BCs
    disp_cons.value = dispRamp(t)

    # Solve the problem
    try:
        (iter, converged) = solver.solve(w)
    except: # Break the loop if solver fails
```

```
        print("Ended Early")
        break

    # Collect results from MPI ghost processes
    w.x.scatter_forward()

    # Write output to file
    writeResults(t)

    # Update DOFs for next step
    w_old.x.array[:] = w.x.array

    # Store  displacement and stress at a particular point  at this time
    timeHist0[ii] = w.sub(0).sub(0).eval([length, length,␣
 ↪length],colliding_cells[0])[0] # time history of displacement
    #
    timeHist1[ii] =  domain.comm.gather(fem.assemble_scalar(shearRxnForce))[0]␣
 ↪# time history of engineering stress

    # Print progress of calculation
    if ii%1 == 0:
        now = datetime.now()
        current_time = now.strftime("%H:%M:%S")
        print("Step: {} | Increment: {}, Iterations: {}".\
              format(step, ii, iter))
        print("      Simulation Time: {} s  of  {} s".\
              format(round(t,4), Ttot))
        print()


# close the output file.
file_results.close()

# End analysis
print("-----------------------------------------")
print("End computation")
# Report elapsed real time for the analysis
endTime = datetime.now()
elapseTime = endTime - startTime
print("-----------------------------------------")
print("Elapsed real time:  {}".format(elapseTime))
print("-----------------------------------------")
```

```
-----------------------------------
Simulation Start
-----------------------------------
Step: Shear | Increment: 1, Iterations: 5
```

```
      Simulation Time: 0.2 s  of  20.0 s

Step: Shear | Increment: 2, Iterations: 5
      Simulation Time: 0.4 s  of  20.0 s

Step: Shear | Increment: 3, Iterations: 5
      Simulation Time: 0.6 s  of  20.0 s

Step: Shear | Increment: 4, Iterations: 5
      Simulation Time: 0.8 s  of  20.0 s

Step: Shear | Increment: 5, Iterations: 5
      Simulation Time: 1.0 s  of  20.0 s

Step: Shear | Increment: 6, Iterations: 4
      Simulation Time: 1.2 s  of  20.0 s

Step: Shear | Increment: 7, Iterations: 4
      Simulation Time: 1.4 s  of  20.0 s

Step: Shear | Increment: 8, Iterations: 4
      Simulation Time: 1.6 s  of  20.0 s

Step: Shear | Increment: 9, Iterations: 4
      Simulation Time: 1.8 s  of  20.0 s

Step: Shear | Increment: 10, Iterations: 4
      Simulation Time: 2.0 s  of  20.0 s

Step: Shear | Increment: 11, Iterations: 4
      Simulation Time: 2.2 s  of  20.0 s

Step: Shear | Increment: 12, Iterations: 4
      Simulation Time: 2.4 s  of  20.0 s

Step: Shear | Increment: 13, Iterations: 2
      Simulation Time: 2.6 s  of  20.0 s

Step: Shear | Increment: 14, Iterations: 4
      Simulation Time: 2.8 s  of  20.0 s

Step: Shear | Increment: 15, Iterations: 4
      Simulation Time: 3.0 s  of  20.0 s

Step: Shear | Increment: 16, Iterations: 4
      Simulation Time: 3.2 s  of  20.0 s

Step: Shear | Increment: 17, Iterations: 4
```

```
      Simulation Time: 3.4 s  of  20.0 s

Step: Shear | Increment: 18, Iterations: 4
      Simulation Time: 3.6 s  of  20.0 s

Step: Shear | Increment: 19, Iterations: 4
      Simulation Time: 3.8 s  of  20.0 s

Step: Shear | Increment: 20, Iterations: 4
      Simulation Time: 4.0 s  of  20.0 s

Step: Shear | Increment: 21, Iterations: 5
      Simulation Time: 4.2 s  of  20.0 s

Step: Shear | Increment: 22, Iterations: 5
      Simulation Time: 4.4 s  of  20.0 s

Step: Shear | Increment: 23, Iterations: 5
      Simulation Time: 4.6 s  of  20.0 s

Step: Shear | Increment: 24, Iterations: 5
      Simulation Time: 4.8 s  of  20.0 s

Step: Shear | Increment: 25, Iterations: 5
      Simulation Time: 5.0 s  of  20.0 s

Step: Shear | Increment: 26, Iterations: 5
      Simulation Time: 5.2 s  of  20.0 s

Step: Shear | Increment: 27, Iterations: 5
      Simulation Time: 5.4 s  of  20.0 s

Step: Shear | Increment: 28, Iterations: 5
      Simulation Time: 5.6 s  of  20.0 s

Step: Shear | Increment: 29, Iterations: 5
      Simulation Time: 5.8 s  of  20.0 s

Step: Shear | Increment: 30, Iterations: 5
      Simulation Time: 6.0 s  of  20.0 s

Step: Shear | Increment: 31, Iterations: 5
      Simulation Time: 6.2 s  of  20.0 s

Step: Shear | Increment: 32, Iterations: 4
      Simulation Time: 6.4 s  of  20.0 s

Step: Shear | Increment: 33, Iterations: 4
```

```
      Simulation Time: 6.6 s  of  20.0 s

Step: Shear | Increment: 34, Iterations: 4
      Simulation Time: 6.8 s  of  20.0 s

Step: Shear | Increment: 35, Iterations: 4
      Simulation Time: 7.0 s  of  20.0 s

Step: Shear | Increment: 36, Iterations: 4
      Simulation Time: 7.2 s  of  20.0 s

Step: Shear | Increment: 37, Iterations: 4
      Simulation Time: 7.4 s  of  20.0 s

Step: Shear | Increment: 38, Iterations: 2
      Simulation Time: 7.6 s  of  20.0 s

Step: Shear | Increment: 39, Iterations: 4
      Simulation Time: 7.8 s  of  20.0 s

Step: Shear | Increment: 40, Iterations: 4
      Simulation Time: 8.0 s  of  20.0 s

Step: Shear | Increment: 41, Iterations: 4
      Simulation Time: 8.2 s  of  20.0 s

Step: Shear | Increment: 42, Iterations: 4
      Simulation Time: 8.4 s  of  20.0 s

Step: Shear | Increment: 43, Iterations: 4
      Simulation Time: 8.6 s  of  20.0 s

Step: Shear | Increment: 44, Iterations: 4
      Simulation Time: 8.8 s  of  20.0 s

Step: Shear | Increment: 45, Iterations: 4
      Simulation Time: 9.0 s  of  20.0 s

Step: Shear | Increment: 46, Iterations: 5
      Simulation Time: 9.2 s  of  20.0 s

Step: Shear | Increment: 47, Iterations: 5
      Simulation Time: 9.4 s  of  20.0 s

Step: Shear | Increment: 48, Iterations: 5
      Simulation Time: 9.6 s  of  20.0 s

Step: Shear | Increment: 49, Iterations: 5
```

```
      Simulation Time: 9.8 s  of  20.0 s

Step: Shear | Increment: 50, Iterations: 5
      Simulation Time: 10.0 s  of  20.0 s

Step: Shear | Increment: 51, Iterations: 5
      Simulation Time: 10.2 s  of  20.0 s

Step: Shear | Increment: 52, Iterations: 5
      Simulation Time: 10.4 s  of  20.0 s

Step: Shear | Increment: 53, Iterations: 5
      Simulation Time: 10.6 s  of  20.0 s

Step: Shear | Increment: 54, Iterations: 5
      Simulation Time: 10.8 s  of  20.0 s

Step: Shear | Increment: 55, Iterations: 5
      Simulation Time: 11.0 s  of  20.0 s

Step: Shear | Increment: 56, Iterations: 4
      Simulation Time: 11.2 s  of  20.0 s

Step: Shear | Increment: 57, Iterations: 4
      Simulation Time: 11.4 s  of  20.0 s

Step: Shear | Increment: 58, Iterations: 4
      Simulation Time: 11.6 s  of  20.0 s

Step: Shear | Increment: 59, Iterations: 4
      Simulation Time: 11.8 s  of  20.0 s

Step: Shear | Increment: 60, Iterations: 4
      Simulation Time: 12.0 s  of  20.0 s

Step: Shear | Increment: 61, Iterations: 4
      Simulation Time: 12.2 s  of  20.0 s

Step: Shear | Increment: 62, Iterations: 4
      Simulation Time: 12.4 s  of  20.0 s

Step: Shear | Increment: 63, Iterations: 2
      Simulation Time: 12.6 s  of  20.0 s

Step: Shear | Increment: 64, Iterations: 4
      Simulation Time: 12.8 s  of  20.0 s

Step: Shear | Increment: 65, Iterations: 4
```

```
      Simulation Time: 13.0 s  of  20.0 s

Step: Shear | Increment: 66, Iterations: 4
      Simulation Time: 13.2 s  of  20.0 s

Step: Shear | Increment: 67, Iterations: 4
      Simulation Time: 13.4 s  of  20.0 s

Step: Shear | Increment: 68, Iterations: 4
      Simulation Time: 13.6 s  of  20.0 s

Step: Shear | Increment: 69, Iterations: 4
      Simulation Time: 13.8 s  of  20.0 s

Step: Shear | Increment: 70, Iterations: 4
      Simulation Time: 14.0 s  of  20.0 s

Step: Shear | Increment: 71, Iterations: 5
      Simulation Time: 14.2 s  of  20.0 s

Step: Shear | Increment: 72, Iterations: 5
      Simulation Time: 14.4 s  of  20.0 s

Step: Shear | Increment: 73, Iterations: 5
      Simulation Time: 14.6 s  of  20.0 s

Step: Shear | Increment: 74, Iterations: 5
      Simulation Time: 14.8 s  of  20.0 s

Step: Shear | Increment: 75, Iterations: 5
      Simulation Time: 15.0 s  of  20.0 s

Step: Shear | Increment: 76, Iterations: 5
      Simulation Time: 15.2 s  of  20.0 s

Step: Shear | Increment: 77, Iterations: 5
      Simulation Time: 15.4 s  of  20.0 s

Step: Shear | Increment: 78, Iterations: 5
      Simulation Time: 15.6 s  of  20.0 s

Step: Shear | Increment: 79, Iterations: 5
      Simulation Time: 15.8 s  of  20.0 s

Step: Shear | Increment: 80, Iterations: 5
      Simulation Time: 16.0 s  of  20.0 s

Step: Shear | Increment: 81, Iterations: 5
```

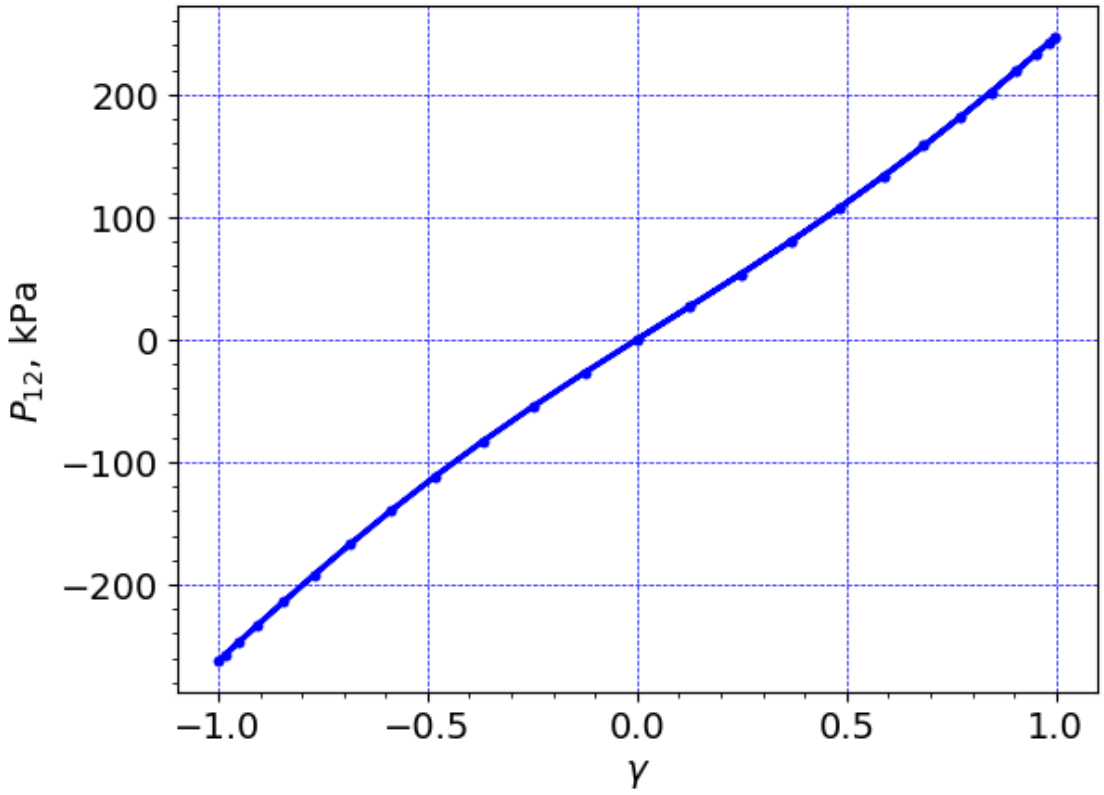
```
      Simulation Time: 16.2 s  of  20.0 s

Step: Shear | Increment: 82, Iterations: 4
      Simulation Time: 16.4 s  of  20.0 s

Step: Shear | Increment: 83, Iterations: 4
      Simulation Time: 16.6 s  of  20.0 s

Step: Shear | Increment: 84, Iterations: 4
      Simulation Time: 16.8 s  of  20.0 s

Step: Shear | Increment: 85, Iterations: 4
      Simulation Time: 17.0 s  of  20.0 s

Step: Shear | Increment: 86, Iterations: 4
      Simulation Time: 17.2 s  of  20.0 s

Step: Shear | Increment: 87, Iterations: 4
      Simulation Time: 17.4 s  of  20.0 s

Step: Shear | Increment: 88, Iterations: 2
      Simulation Time: 17.6 s  of  20.0 s

Step: Shear | Increment: 89, Iterations: 4
      Simulation Time: 17.8 s  of  20.0 s

Step: Shear | Increment: 90, Iterations: 4
      Simulation Time: 18.0 s  of  20.0 s

Step: Shear | Increment: 91, Iterations: 4
      Simulation Time: 18.2 s  of  20.0 s

Step: Shear | Increment: 92, Iterations: 4
      Simulation Time: 18.4 s  of  20.0 s

Step: Shear | Increment: 93, Iterations: 4
      Simulation Time: 18.6 s  of  20.0 s

Step: Shear | Increment: 94, Iterations: 4
      Simulation Time: 18.8 s  of  20.0 s

Step: Shear | Increment: 95, Iterations: 4
      Simulation Time: 19.0 s  of  20.0 s

Step: Shear | Increment: 96, Iterations: 5
      Simulation Time: 19.2 s  of  20.0 s

Step: Shear | Increment: 97, Iterations: 5
```

```
      Simulation Time: 19.4 s  of  20.0 s

Step: Shear | Increment: 98, Iterations: 5
      Simulation Time: 19.6 s  of  20.0 s

Step: Shear | Increment: 99, Iterations: 5
      Simulation Time: 19.8 s  of  20.0 s

Step: Shear | Increment: 100, Iterations: 5
      Simulation Time: 20.0 s  of  20.0 s

----------------------------------------
End computation
----------------------------------------
Elapsed real time:  0:03:12.366679
----------------------------------------
```

14 Plot results

[19]:
```
# set plot font to size 14
font = {'size'   : 14}
plt.rc('font', **font)


# Only plot as far as we have time history data
ind = np.argmax(timeHist0)
#
fig = plt.figure()
ax=fig.gca()

# plot figure
plt.plot(timeHist0[:ind]/length, timeHist1[:ind], c='b', linewidth=2.0,␣
 ↪marker='.')
#-------------------------------------------------------------
#ax.set.xlim(-0.05,0.05)
#ax.set.ylim(-0.03,0.03)
#plt.axis('tight')
plt.grid(linestyle="--", linewidth=0.5, color='b')
ax.set_xlabel(r'$\gamma$',size=14)
ax.set_ylabel(r'$P_{12}$, kPa ',size=14)

#ax.set_title("Shear stress-strain curve", size=14, weight='normal')
from matplotlib.ticker import AutoMinorLocator,FormatStrFormatter
ax.xaxis.set_minor_locator(AutoMinorLocator())
#ax.xaxis.set_minor_formatter(FormatStrFormatter("%.2f"))
ax.yaxis.set_minor_locator(AutoMinorLocator())
plt.show()
```

```
fig = plt.gcf()
fig.set_size_inches(7,5)
plt.tight_layout()
plt.savefig("results/3D_finite_elastic_simple_shear_fenicsX.png", dpi=600)
```




```
<Figure size 700x500 with 0 Axes>
```