

# ME 513 HMWK 5

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**7.1.4 A simple source of sound in air radiates an acoustic power of 10 mW at 400 Hz. At 0.5 m from the source, compute**

**(a) the intensity,** The intensity is given by equation (7.2.15)

$$I = \frac{1}{8} \rho_0 c (Q/\lambda r)^2$$

We can find  $Q$  from the equation for acoustic power (7.2.16)

$$\begin{aligned} \Pi &= \frac{1}{2} \pi \rho_0 c (Q/\lambda)^2 \\ &= \frac{1}{2} \pi \rho_0 c \frac{Q^2}{c^2/f^2} \\ Q &= \sqrt{\frac{2\Pi c}{\pi \rho_0 f^2}} \end{aligned}$$

which gives

$$Q = \sqrt{\frac{2(1. \times 10^{-2} \text{ W})(343 \text{ m/s})}{\pi (1.275 \text{ kg/m}^3)(400 \text{ Hz})^2}} = 3.27 \times 10^{-3} \text{ m}^3/\text{s}$$

and so, the intensity is

$$I = \frac{1}{8} (1.275 \text{ kg/m}^3) (343 \text{ m/s}) \left[ \frac{(3.27 \times 10^{-3} \text{ m}^3/\text{s})(400 \text{ Hz})}{(343 \text{ m/s})(0.5 \text{ m})} \right]^2 = 3.16 \text{ mW/m}^2$$

**(b) the pressure amplitude,** The pressure amplitude is given by equation 7.2.14

$$P = \frac{1}{2} \rho_0 c (Q/\lambda r) = \frac{1}{2} (1.275 \text{ kg/m}^3) (343 \text{ m/s}) \frac{(3.27 \times 10^{-3} \text{ m}^3/\text{s})(400 \text{ Hz})}{(343 \text{ m/s})(0.5 \text{ m})} = 1.663 \text{ Pa}$$

**(c) the particle speed amplitude,** Using linear acoustics, we have that

$$I = \frac{1}{2} P U$$

and therefore

$$U = \frac{2I}{P} = \frac{2 \cdot 3.16 \text{ mW/m}^2}{1.663 \text{ Pa}} = 3.8 \text{ mm/s}$$

**(d) the particle displacement amplitude, and** Finally, the displacement amplitude is another time integral from the particle speed, so

$$\Xi = \frac{U}{f} = \frac{3.8 \text{ mm/s}}{400 \text{ Hz}} = 9.5 \mu\text{m}$$

(e) **the condensation amplitude.** The condensation is define as the pressure divided by the density times the speed of sound squared

$$S = \frac{P}{\rho_0 c^2} = \frac{1.663 \text{ Pa}}{(1.275 \text{ kg/m}^3)(343 \text{ m/s})^2} = 1.1 \times 10^{-5}$$

**7.4.1 For a baffled piston of radius  $a$  driven at angular frequency  $\omega$ ,**

(a) **find the smallest angle  $\theta_1$  for which the pressure is zero in the far field,** To find the angle  $\theta_1$  desired, we use the smallest node given by

$$ka \sin \theta_m = j_{1m} \quad m = 1, 2, 3, \dots$$

so

$$\begin{aligned} ka \sin \theta_1 &= j_{11} \\ \sin \theta_1 &= \frac{j_{11}}{ka} \\ \theta_1 &= \arcsin \left( \frac{j_{11}}{ka} \right) = \arcsin \left( \frac{j_{11}c}{\omega a} \right) \end{aligned}$$

(b) **find the greatest finite distance for which the pressure is zero on the acoustic axis, and** From the equation (7.4.4), we have

$$\tilde{p}(r, 0, t) = \rho_0 c u_0 \left\{ 1 - \exp \left[ -jk \left( \sqrt{r^2 + a^2} - r \right) \right] \right\} \exp [j(\omega t - kr)]$$

which has the magnitude

$$p(r, 0) = 2\rho_0 c u_0 \left| \sin \left\{ \frac{1}{2}kr \left[ \sqrt{1 + (a/r)^2} - 1 \right] \right\} \right|$$

and the extrema of this is at

$$\frac{1}{2}kr \left[ \sqrt{1 + (a/r)^2} - 1 \right] = \frac{m\pi}{2} \quad m = 0, 1, 2, \dots$$

therefore, moving down from large  $r$ , we can find the first minima,  $r_2$  at

$$\begin{aligned} \frac{r_2}{a} &= \frac{a}{2\lambda} - \frac{\lambda}{2a} \\ r_2 &= \frac{a^2}{2\lambda} - \frac{\lambda}{2} \end{aligned}$$

and with

$$\begin{aligned} k &= \frac{\omega}{c} \\ \lambda &= \frac{2\pi}{k} = \frac{2\pi c}{\omega} \end{aligned}$$

so

$$r_2 = \frac{a^2\omega}{4\pi c} - \frac{\pi c}{\omega}$$

(c) **discuss the possibility of obtaining  $\theta_1 \ll 1$  and  $r_1/a \ll 1$  simultaneously.** At very large  $r_1/a$ , we approach a monopole, and as such,  $\theta_1 \rightarrow \frac{\pi}{2}$ . This means that  $\theta_1 \ll 1$  only when  $r_1/a \gg 1$  (or when we're in the very dipole like region).

**7.4.6C**

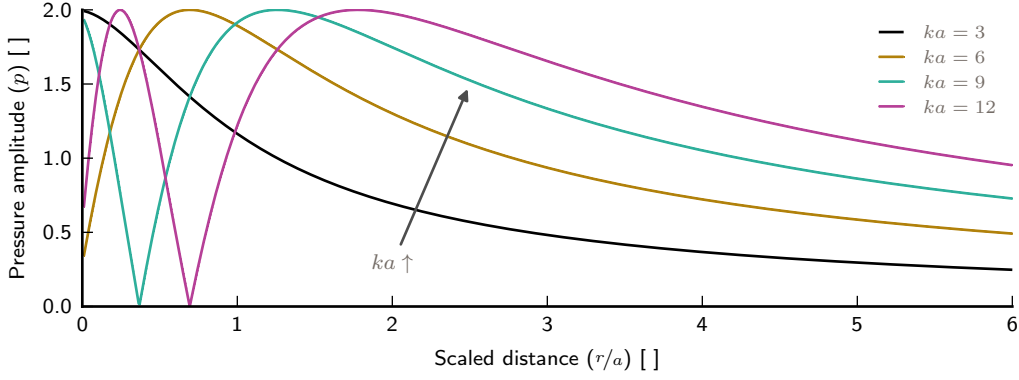


Figure 1: Plot of pressure amplitude with scaled distance for different values of  $ka$

**(a) For a circular piston, plot the on-axis pressure amplitude as a function of scaled distance  $r/a$  for several values of  $ka$  between 3 and 12.** Using equation (7.4.5), we have

$$p(r, 0) = 2\rho_0 c u_0 \left| \sin \left\{ \frac{1}{2} k r \left[ \sqrt{1 + (a/r)^2} - 1 \right] \right\} \right|$$

and we can convert this into the quantity  $ka$  as

$$p(r, 0) = 2\rho_0 c u_0 \left| \sin \left\{ \frac{1}{2} \frac{ka}{a} r \left[ \sqrt{1 + (a/r)^2} - 1 \right] \right\} \right|$$

$$p(r/a, 0) = 2\rho_0 c u_0 \left| \sin \left\{ \frac{1}{2} ka (r/a) \left[ \sqrt{1 + (r/a)^{-2}} - 1 \right] \right\} \right|$$

The solutions to this with  $ka = 3, 4, 5, \dots, 12$  are shown in figure 1.

**(b) Plot the range beyond which the pressure amplitude is within 10% of the asymptotic form (7.4.7).** With the asymptotic form of

$$p(r) = \frac{1}{2} \rho_0 c u_0 (r/a)^{-1} ka$$

we can find where that reaches 90% of the true form by solving the equation

$$2\rho_0 c u_0 \left| \sin \left\{ \frac{1}{2} ka (r/a) \left[ \sqrt{1 + (r/a)^{-2}} - 1 \right] \right\} \right| = 0.9 \cdot \frac{1}{2} \rho_0 c u_0 (r/a)^{-1} ka$$

$$4.44 = \frac{(r/a)^{-1} ka}{\left| \sin \left\{ \frac{1}{2} ka (r/a) \left[ \sqrt{1 + (r/a)^{-2}} - 1 \right] \right\} \right|}$$

which, solved numerically, occurs at  $r/a = 4.26$  for a  $ka = 12$ . Figure 2 shows this region on a plot of scaled distance.

**(c) For a piston of 20 cm radius operating at 4 kHz in water, find the distance corresponding to (b).** First, we must determine the  $r/a$  for asymptotic behavior with the given  $ka$ .

$$k = \frac{2\pi}{\lambda} = \frac{2\pi f}{c} = \frac{2\pi \cdot 4 \times 10^3 \text{ Hz}}{343.2 \frac{\text{m}}{\text{s}}}$$

so,

$$ka = 366.153$$

and solving numerically, the asymptotic region starts at

$$r/a = 122.23$$

which is

$$r = 24.4 \text{ m}$$

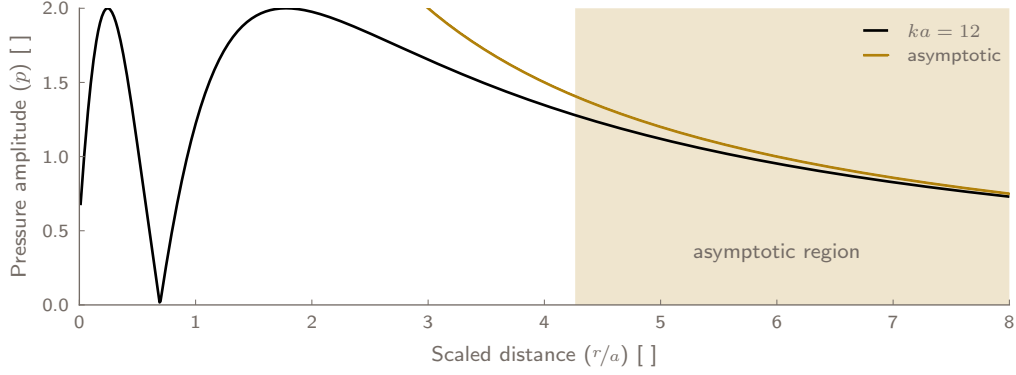


Figure 2: Asymptotic region highlight on plot of pressure amplitude with scaled distance for a single value of  $ka$

### 7.5.1

(a) Find the resonance frequency of a piston transducer with the mechanical properties  $m$ ,  $s$ , and  $R_m$  radiating into a fluid with specific acoustic impedance  $\rho_0 c$ . Assume  $ka \gg 2$ . Using equation (7.5.11), the impedance is

$$z_r = \rho_0 c S [R_1(2ka) + jX_1(2ka)]$$

and well above  $ka = 2$ , we have the approximations

$$\begin{aligned} kX_1(2ka) &\rightarrow \frac{2/\pi}{ka} = \frac{2}{\pi ka} \\ R_1(2ka) &\rightarrow 1 \\ z_r &\rightarrow R_r \approx S\rho_0 c \end{aligned}$$

therefore

$$z_r = \rho_0 c S \left[ 1 + j \frac{2}{\pi ka} \right]$$

and comparing that to the form of the mechanical impedance

$$z_m = R_m + j \left( \omega m - \frac{s}{\omega} \right)$$

we can split the radiation impedance into parts

$$\begin{aligned} z_m + z_r &= (R_m + \rho_0 c S) + j \left( \frac{2\omega m}{\pi ka} - \frac{2s}{\pi \omega ka} \right) \\ &= (R_m + \rho_0 c S) + j \left( \frac{2cm}{\pi a} - \frac{2cs}{\pi \omega^2 a} \right) \end{aligned}$$

and the resonance frequency is where

$$\text{Im} \{z_m + z_r\} = 0$$

which is

$$\begin{aligned} \frac{2\omega m}{\pi ka} - \frac{2s}{\pi \omega ka} &= 0 \\ \frac{2\omega m}{\pi ka} &= \frac{2s}{\pi \omega ka} \\ \omega^2 &= \frac{s}{m} \\ \omega &= \sqrt{\frac{s}{m}} \end{aligned}$$

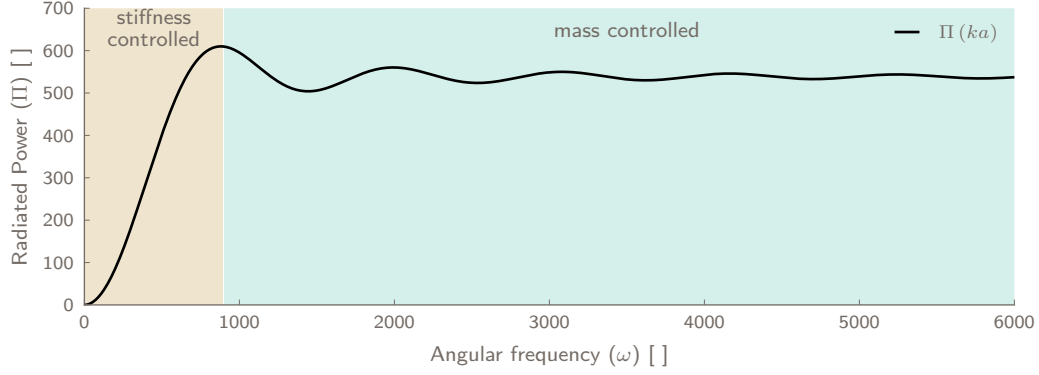


Figure 3: Power radiated as a function of frequency with mass and stiffness controlled regions labeled

(b) Sketch the frequency dependence of the radiated power if the transducer is driven with a force of constant amplitude. Assume that the resonance frequency occurs well above the lower limit of the approximations implicit in  $ka \gg 2$ . Indicate where the transducer is mass controlled and where it is stiffness controlled. The radiated power is given by

$$\pi = \frac{1}{2} u_0^2 R_r$$

and assuming along the axis, we have

$$\begin{aligned} \Pi &= \frac{1}{2} u_0^2 R_r \\ &= \frac{1}{2} u_0^2 \rho_0 c S \cdot \operatorname{Re} \left\{ \left[ 1 - \frac{2J_1(2ka)}{2ka} \right] + j \left[ \frac{2H_1(2ka)}{2ka} \right] \right\} \end{aligned}$$

and this is shown in figure 3. The mass controlled region occurs when

$$z_m + z_r \approx j \frac{2cm}{\pi a}$$

or when the frequency is high (on the plateau of the chart). The stiffness controlled region occurs when

$$z_m + z_r \approx -j \frac{2cs}{\pi \omega^2 a}$$

or in the increase part of the chart.