Muss Courtinuity Eg

Hilbert General Belonce Equation 4 -> property to be belonced [4dV ->total amount in V J - Flux property across surface - \$ J-ids -> net flow across surface 7 -> generation of 7 per unit volume { Change of 4} = { Flux across} + { generation} Dt Sv. 42V=- & J. Ads + Sty dv using Reynolds and green's - & J·ils=- S. マ· JLV we get 27+ V. (40)=-V. J+ 4

Use Hilbert Greneral Balance

$$\frac{\partial \nabla \nabla}{\partial t} + \nabla (\nabla \nabla \nabla) = -\nabla P - \nabla \cdot \mathbf{1} + C_{1} = \frac{1}{2}$$

Cylindrical

Momentum Eq (Nut. Circ)

Start from momentum to (use conservative form: Vp V not p(V-V)

Assumption Pensity change due to thermal expansion, but it's only important in growity terms

Use thered expension wefficient B

they if is small (hydrostatic)

Kinetic Energy Eg

Use Hilbert General Belance Equation

\[\vec{V} \cdot \big| \big| \frac{\text{trunsform}}{\text{from phomentum}} \big| \big|

Hilbert Greneral Balance Equation

$$\frac{1}{4} = \rho(u + \frac{\vec{v}^2}{3})$$
T.E. K.E.

$$\frac{1}{3} = \vec{v} + \vec$$

Thermal Energy Equation

$$\begin{bmatrix} E E \end{bmatrix} - \begin{bmatrix} K E \end{bmatrix}$$

$$\frac{dp(u+\frac{\sqrt{3}}{2})}{dt} + \nabla \cdot \left[p(u+\frac{\sqrt{3}}{2})\vec{v} \right] = -\nabla \cdot \vec{g} - \nabla \cdot (p\vec{v}) + \nabla \cdot (\vec{u} \cdot \vec{v}) + p\vec{v} \cdot \vec{g} + \vec{g}$$

Pu = -Vig - pv. v for most cases
(small dissipation, no internal heat gen)

Two Component Mixture

Hilbert Greneral Belance Equation $\mathcal{A} = \mathcal{A}_{k}$ $\mathcal{A} = \mathcal{A}_{k}$ $\mathcal{A} = \mathcal{A}_{k}$ $\mathcal{A}_{g} = r_{k}$ So $\frac{\partial \mathcal{A}_{k}}{\partial t} + \nabla \mathcal{A}_{k} \nabla = -\nabla \cdot \mathcal{A}_{k}$ $\mathcal{A}_{g} = r_{k}$ Change mass convection mass flux

where flux

Hilbert GBE

 $\frac{\partial \vec{v}}{\partial t} + \nabla p \vec{v} \vec{v} = -\nabla p - \nabla \cdot \vec{v} - \nabla \Sigma P_k \vec{V}_{km} \vec{V}_{km} + \mathcal{E}_{R} \vec{g} \qquad [Mixture ME]$ 14:16e + GBE $4 = p(u + \frac{\vec{v}^2}{2})$

J=TT·V; Zy=EpkVk·g+8k Types

State Ex Examples -> Ideal Gas (p=RTP, u=u(t)), Incomp (p(pT)=p) Mechanical

Inviscial (88 % correct) -> TE=0

Linearly viscous -> Tyx = zu dy

could be viscous force, heat flux, mass flux diffesion

Thermal

heat conduction - 3 =- KVT

internal heat you - g=q(r,T) fission, electrical resostance

Mass Diffusion

Mass diffusion flux in = R(V-V)

Diffusion model specified by constituitive ex

in = Pr(vi-v)=-pDV(===>DVWk

Entropy generation whick

Tols = du - Bolp

DST DE + DE DES

PS = V. 8 1 VV + 8

1 + V. (=) - = = 1>0

d= 8 1/2 - 1/2 20

Too, -8. VIZO; -I : WZO

Grow high to Liss potion

so satisfies entropy generation

Falling Luminur Film

Assumptions

B.C. Vx =0 at x=t (velocity of well cannot be finite) => Vx =0 everywhere

$$P\left\{\frac{\partial y}{\partial x} + v_{y}\frac{\partial y}{\partial x} + v_{y}\frac{\partial y}{\partial y} + v_{z}\frac{\partial y}{\partial z}\right\} = -\frac{\partial p}{\partial x} + \mu_{z}\left\{\frac{\partial y}{\partial x} + \frac{\partial y}{\partial y} + \frac{\partial y}{\partial z}\right\} + p\frac{\pi}{3}$$

$$-\frac{\partial p}{\partial x} + p\frac{\pi}{3} = 0$$

$$p g_{x} = p\cos\theta g$$

B.C. at x=0, no short =
$$\frac{\partial V_{z}}{\partial x} = 0$$

B.C. at x=8, $V_{z} = 0$

$$v_2 = \frac{p_9 \, 8^7 \cos \alpha}{2 \pi} \left[1 - \left(\frac{\kappa}{8^3} \right)^7 \right] \implies 1$$

1) Scaling parumeter

Distance

Velocity
V= V

Pressure

P*= P-B

BVR

temperature T*= T-To

Density PK=P

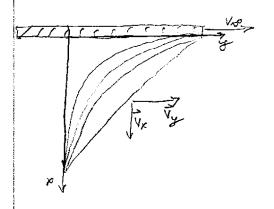
$$Re = \frac{P_0 VD}{M} = \frac{P V^2/D}{M(VD)/D} = \frac{Inertia}{Viscous force}$$

$$P_r = \frac{T}{a} = \frac{MP}{M/DCV} = \frac{Inertia}{Inertia} \frac{Viscous force}{Missingly}$$

$$E_C = \frac{V^2}{CPDT} = \frac{P V^2/D}{CPDT/D} = \frac{K.E.}{Inertia} \frac{Convection}{Inertia}$$

$$F_r = \frac{V^2}{3D} = \frac{P V^2/D}{Pg} = \frac{Inertia}{gravity}$$

$$P_e = Re P_r$$



Assumptions

AD No 2 Dependence (\$\frac{1}{2} = 0)

No velocity in =-dir (\$v_2 = 0)

et t<0, \$v_2 = 0, \$v_3 = 0, \$t_3 = 0, \$v_4 = 0, \$v_5 =

$$\frac{C.E.}{dt} \circ + \nabla \cdot p \stackrel{?}{\nabla} = 0, \quad \nabla \cdot \vec{v} = 0$$

$$\frac{\partial V}{\partial x} + \frac{\partial V}{\partial y} = 0$$

$$\frac{\partial V}{\partial x} + \frac{\partial V}{\partial y} = 0$$

$$\frac{\partial V}{\partial x} = 0$$

$$\frac{\partial V}{\partial x} = 0$$

$$V_{E} = 0$$

$$V_{E} = 0$$

M.E.

$$\frac{M.E.}{S} = \frac{SP}{SV} + V_{S} = \frac{S^{2}V_{Y}}{SV} + V_{S} = \frac{S^{2}V_{Y}}{SV} + \frac{S^{2}V_{Y}}{SV$$

becomes similar w/ hast conduction through rod

700

Energy Eq

$$P(\sqrt{\frac{1}{3}} + \sqrt{\frac{1}{3}}) = k \sqrt{2} + q + p \sqrt{2} - \frac{1}{3} + p \sqrt{2} + \frac{1}{3} + \frac{1}$$

Thermal Penetrution Depth

$$\frac{\partial T}{\partial t} = q \frac{\partial^{2}T}{\partial x^{2}}$$

$$O = T - T_{op}$$

$$\frac{\partial A}{\partial t} = q \frac{\partial^{2}A}{\partial x^{2}}$$
and
$$O(x, 0) = T_{o} - T_{op} = Q_{o}$$
one
$$O(x, t) = Q_{o}$$

$$U_{o} = \frac{d^{2}Q}{dx^{2}}$$

$$U_{o} = \frac{d^{2}Q}{dx^$$

Time Averaged Turbulence

$$\mathcal{L}^{t} = \rho V' V' \\
\mathcal{L}^{M} = \mathcal{I} \left[\nabla \vec{V} + (\nabla \vec{V})^{t} \right] \\
\mathcal{L}^{T} = \mathcal{L}^{M} + \mathcal{L}^{t}$$

$$\frac{\partial \rho \vec{V}}{\partial t} + \nabla \rho \vec{V} \vec{V} = -\nabla \rho - \nabla \cdot \mathcal{L}^{T} + \rho \vec{g} \qquad \left[\text{Turbulent M.E.} \right]$$

$$\rho C_{V} \frac{\partial \vec{T}}{\partial t} = \left[k \nabla^{2} T - \nabla \cdot \rho c_{V} T' V' \right] + g' \qquad \left[\text{Turbulent E.E.} \right]$$

Velocity Profiles

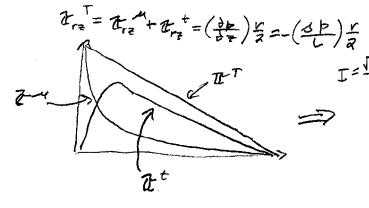
turbulent v=Vmn, [1- \frac{\forall}{2}]/7

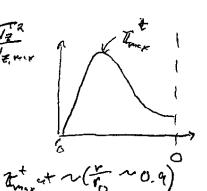
Lavairer V=Vm, [1-(Vm,)]

stress in turbulent pipe flow

steady state No Grevity Fully developed Axisymmetric

ク(ませんできょく dv + v dv + v dv + v dv)=- カラ + Po + [ナカー(アス)+ナ カスナ カスナ カスナ サーナナ(アマルナ)





- 1) -7 Fluid clement moves but doesn't transfer any momentum until it has traveled the entire length
 - momentum r goined by sm $\begin{cases} x \cdot comp \\ your enfund \\ your enfund \\ you fund by \end{cases} = 8 m 8 V_{x};$ $\begin{cases} x \cdot comp \\ you fund \\ y \cdot comp \end{cases} = \frac{9m8 V_{x}}{5t}$ $\begin{cases} x \cdot comp \\ y \cdot comp \\ y \cdot comp \end{cases} = \frac{9m8 V_{x}}{5t}$ $\begin{cases} x \cdot comp \\ y \cdot comp \\ y \cdot comp \end{cases} = \frac{8m}{5t} = \frac{8m}{5t} \cdot \frac{8m$

$$\frac{\delta v_x = \lambda \frac{\delta \overline{v}_x}{\delta y} e}{\frac{1}{A} \frac{\delta m}{\delta t} = \rho |v_y|} \Rightarrow \mathcal{X}^t = -2 |v_y| \frac{d\overline{v}_x}{dy}$$

(3) vg x v,

1/21 = K, Vx 1

Introduce non dimensionalized parameters

v*= \frac{\frac{1}{\text{tap}}}{\text{tap}}, \quad y*= \frac{\frac{1}{\text{tap}}}{\text{T}}

\frac{\dv*}{\dy*} = \frac{1}{\text{ty}} \quad \frac{\pi}{\text{T}} = -\frac{1}{\text{kln}} \quad \frac{\pi}{\text{tap}} + C_1

so 3 regions

- 1) Laminer subleyer [7=0 = V*=y* for y* 45]
- 2) Buffer Liger [v=-3.05+5lng* for 5=4×50]
- 3) Turbelent love [v=5.5 + 2.5 en y* for y*>30]

