

# **Customizable BioMechanical Orthotic Knee Joint for Exoskeletons Thesis**

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## **Abstract**

Current research demonstrates that the shank (lower leg) linearly extends as the knee flexes, in a roughly quartic trajectory. However, most actuated orthotics do not consider this tibiofemoral relationship. This thesis presents a customizable biomechanical orthotic knee joint for medical rehabilitation which can follow this relationship. The design is easily manufacturable with common machining tools and FDM 3D printing techniques. Most importantly, it is customizable to each patient with the simple replacement of one component. Finally, this thesis will present a software workflow and tools to identify this tibiofemoral relationship in patients using motion capture technologies.

# **Chapter 1**

## **Introduction**

test

# **Chapter 2**

## **Background**

### **2.1 Paraplegia and Rehabilitation**

Paraplegia is a medical term used to define where a patient loses feeling and/or movement in their lower two limbs. In comparison, quadriplegia (also sometimes known as tetraplegia) is the loss of control in all four limbs. It is important to note that not all feeling/movement needs to be lost in order for someone to be considered paraplegic [5]. Only 30% of all paraplegic and quadriplegic patients are considered complete lesions, where there is no sensation and no mobility in the lower limbs [12].

Paralysis is usually caused by trauma, such as sports injuries, vehicle accidents, or accidental falls, when the spine gets injured (see Figure 2.1). However, it can also be caused by specific diseases, including multiple sclerosis, amyotrophic lateral sclerosis, stroke, and in specific cases cancer [19]. Common effects of

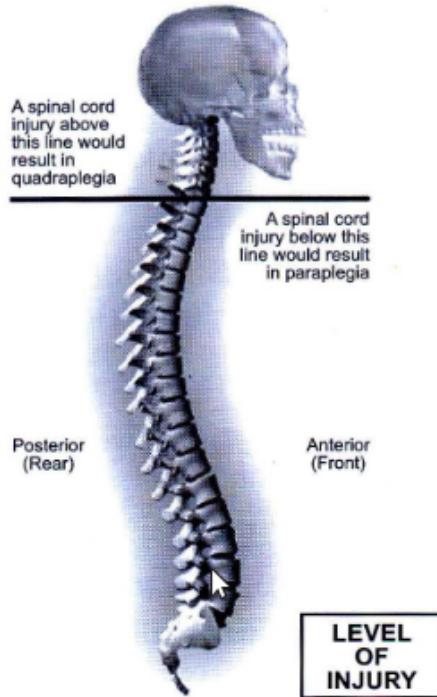


Figure 2.1: Location of spinal chord injury will determine type of paralysis [12]

paraplegia include:

- Loss of mobility, reflexes, and sensation
- Muscular weakness and atrophy
- Hormonal variations
- Gastrointestinal and bowel/bladder problems
- Muscle spasms
- Reduced cardiorespiratory fitness and increased likelihood of cardiorespiratory issues

**TODO:** Talk about skin problems in paralysis patients

Rehabilitation can play a key role in reducing these side effects in patients who experience paraplegia. Mainly, physical therapy for paralysis patients focus on three main types of exercises: stretching, strengthening, and aerobic. Additionally, paralysis patients may go through gait training with the assistance of medical devices.

### 2.1.1 Physical Therapy for Paralysis Patients

#### Stretching

Stretching is considered one of the most important exercises [12], more-so than any other form of exercise because it can be done often and at home. Carefully designed exercises (like seen in Figure 2.2) can improve flexibility, reduce muscle spasms, reduce the chance of injury, and relieve contractures [18] [9] [20]. Some common stretches include bilateral adductor stretches, quadriceps stretches, and hip flexor stretches.

#### Cardiorespiratory and Cardiovascular Training

Due to the difficulty of exercise, cardiovascular and cardiorespiratory activities are also very important to maintain health in paralysis patients. Aerobic exercises can increase energy levels, improve lung and heart function, control body weight, and reduce fatigue [12] [6]. A study showed that patients who suffer from neuromuscular deficiencies such as paraplegia suffered decreasing  $\text{VO}_2 \text{ max}$  compared

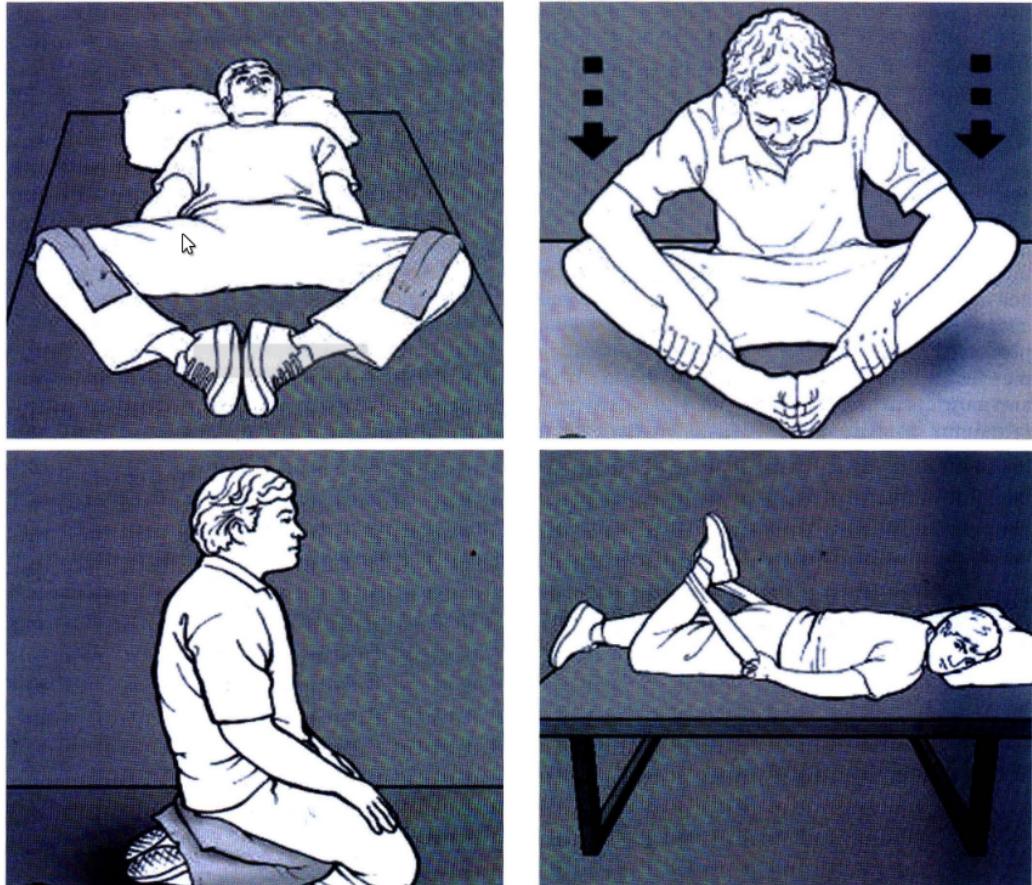


Figure 2.2: Bilateral Adductor Stretches (top) and Quadriceps Stretches (bottom) for paraplegic and quadriplegic patients [12]

to control subjects with no issues [6]. VO<sub>2</sub> max a common metric that measures the maximum rate of oxygen utilization during heavy exercise. Combination of the upper body and lower body in paraplegic patients can strengthen the paralyzed limbs while also activating healthy limbs. Some researchers have even proposed introducing wheelchair racing as a sport in an effort to help with rehabilitation after paraplegia [17].

## **Strength Training**

Improving strength in muscles may actually partially reverse the loss of mobility in partially paralyzed patients, while also improving muscle tone [6] and preventing bone atrophy [18]. This type of exercise can be split into two major regions: training of affected limbs and muscles, and the training of non-affected regions. Affected limbs can benefit from an increase in mobility and definition, and can generally reduce the likelihood of muscular atrophy. Additionally, strong hip and leg muscles in partially paraplegic patients can help in gait training and increase the possibility of usage in life. On the other side, increasing or maintaining strength in unaffected regions can help with quality of life improvement. Often, paraplegic patients may elect to use crutches or canes as an assisted mobility device in the real world. Increasing arm/shoulder strength and endurance will also increase capability for patients to use some of these assisted devices. Finally, back and abdomen muscles are very important to strengthen to maintain posture and improve gait performance [3].

## **Hydrotherapy**

Hydrotherapy (exercising in water) is a notable way for patients suffering from paraplegia to better strengthen muscles and improve cardiovascular health. Due to similar buoyancy, water can reduce the effects of gravity without any external assistive devices. At the same time, the increased density of the water (in comparison to air) creates a natural resistance without the use of weights or elastics.

Therefore, hydrotherapy is used in paraplegic patients to increase muscle power, increase endurance, and even help with gait training (see subsection 2.1.2). In minor cases of paralysis, some patients even use swimming as a way to exercise [12] [22].

## 2.1.2 Gait Training

Gait training has become the best way to improve motor functions in those who have partially or fully lost mobility in their legs and torso. The premise of this exercise is to have patients do similar movements to what one would do without their disability, like walking and climbing stairs. Essentially, the goal is to help the patient relearn the gaits they previously knew. Spinal neuronal circuits degrade quickly - in just a year, they can lose most of their potency, essentially unlearning any gait abilities the patient had in the past [7] [12] [3]. Gait training can help reconnect the broken spinal neurons, and improve motor function and balance in a patient. In fact, several studies have shown that some patients with full spinal chord injuries have been able to recover part or even all of their walking capabilities through gait training [7] [23]! <sup>1</sup>

### Use of Assistive Devices for Gait Training

Since most patients suffering from paralysis won't be able to hold themselves up, there have been many different proposals to compensate for gravity. At lower

---

<sup>1</sup>There is significant research in the benefits of gait training for paraplegic and quadriplegic patients. Not all prior work is cited here.

levels of paralysis, canes, walkers, and other walking assisted devices can help. Hydrotherapy has also been used with gait training due to the similar densities of humans and water [22]. With more serious cases of paralysis, robotic solutions and other active orthotics have been proposed and used in clinical settings.

Standard solutions like canes and walkers will only work for patients with mild paralysis. Canes are designed to support only 25% of body weight [12]. They can also be fairly unstable, since they usually only have at most 4 points of contact with a very small ground contact area. Walkers are better than canes, since they can support up to 50% of body weight [12]. However, canes, walkers, and crutches have one downside: the required upper-body strength. Mild lower-limb paralysis cases usually can benefit from these inexpensive tools to help with gait training. However most patients will struggle holding themselves up during gait training.

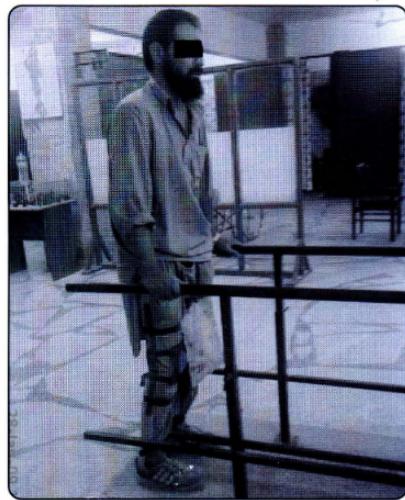


Figure 2.3: Patient using basic splint orthosis [12]

Orthosis are the next level up in assistive devices. They can come in many different shapes and can be designed to fit a patient's progress. At the lowest levels are specialty splints or braces (seen in Figure 2.3) that can help keep joints locked or reduce load of a joint through passive springs. These solutions often cost very little in material, and apply normal loads on the user's skeletal system - helping to prevent bone atrophy. Actively powered orthosis also exist with various levels of research and clinical trials (see section 2.3), and can be separated in two major groups.

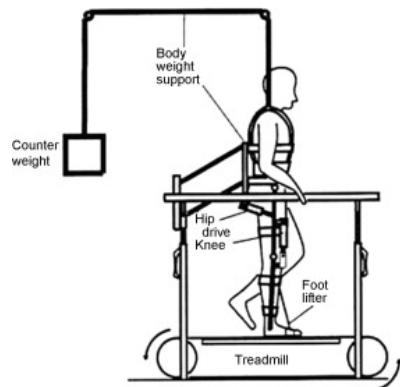


Figure 2.4: Comparison between an exoskeleton with active gravity compensation (left) and an exoskeleton with passive gravity compensation (right) [7]

Actively compensating exoskeletons (see left image in Figure 2.4) are orthosis devices that use various types of actuators, sensors, and gait controllers to help keep patients standing and walking with little to no strength required (from the patient). These types of exoskeletons use up a significant amount of energy, since they must essentially do all the physical work that leg muscles would normally do.

**TODO:** Add additional figure of an active GCS exo

This usually means very powerful actuators and motors with precise and stable control, and large batteries (which add to overall weight) or a large/long tether. Such power increases the overall flexibility of the system, however, at a cost. It also increases complexity of the control software, and introduces a safety risk of attaching powerful actuators to patient limbs.

To mitigate some of these risks, some solutions separate the exoskeleton and the gravity compensation (see right image in Figure 2.4). A separate mechanical system supports the weight of the user usually through a counterweight system or a gantry of some sort. This allows for the actuator in the exoskeleton orthosis to be weaker or power (current) limited to prevent injury in case of a malfunction. However, such systems are much more limited in their uses, since the power is hardware limited and more infrastructure is needed to use the device. Additionally, any actively compensating exoskeleton orthosis can be current limited and be used with a mechanical gravity compensation system.

However, one of the biggest struggles with orthoses is the dermatological problems that they may cause. As explained earlier, patients suffering from paralysis are at an increased risk of developing skin complications [8]. This can be at least partially attributed to reduced sensitivity in paralyzed areas of the body. Therefore, any assisted devices that attach to a patient's skin should accurately follow the natural trajectory of the skin to avoid unnecessary rubbing.

## Robotics for Paralysis Rehabilitation

Robotics have a very large opportunity to improve the rehabilitation process in paraplegic patients. Clinical research in using robotic orthosis for gait training and other rehabilitation exercises show positive improvement in most patients when considering quality of life [11] [2].

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## 2.2 Human Knee Model

The human knee was initially considered as a pin joint, but research as early as 1992 suggested differently [15]. This study showed that the joint does extend as it bends by attaching motion capture markers to the femur and tibia of 5 different subjects. It also suggested that there was a difference between loaded and unloaded knees.

Several studies since then have confirmed a non-linear knee flexion and extension relationship, as well as the difference between loaded and unloaded knees. Instead of intrusive motion capture markers, modern Magnetic Resonance Imaging (MRI) and advanced motion capture systems [14] has allowed for more precise bone tracking for both unweighted and weighted knee joints, since a 3 dimensional model can be built of the joint. A study performed by Iwaki *et. al* using MRI data of cadaver knee joints found that the femur posterior circular arc is responsible for the linear extension of the leg through the flexion process, similar to the movement of a mechanical cam (see Figure 2.5) [13]. Through the bending process, the joint didn't rotate much outside the plane of flexion. These results

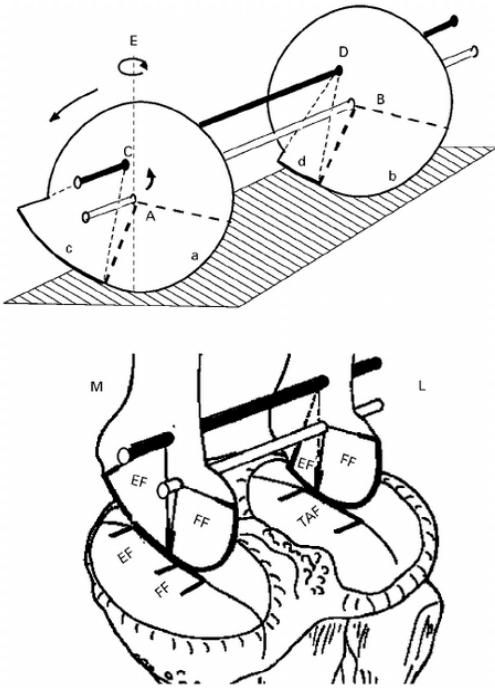


Figure 2.5: Diagram from [13] depicting the internal anatomy of a knee with respect to the movement patterns. The movement characteristics can be closely related to a cam mechanism

were further confirmed during the dissection of the cadavers as well as in the followup research with live human knee joints [10]. Additionally, the research also showed tibiofemoral motion moved forward roughly 4mm when the knee joint was loaded.

This research was taken a step further with the parameterization of a knee joint's flexion and extension. The goal was to define a knee joint model to better create artificial mechanisms for rehabilitation exoskeletons. Based on MRI data of unloaded knees from [13], ellipses were fitted to the ends of the tibia and femur to approximate the relationship between the distance  $r$  and flexion angle  $\theta$ . The

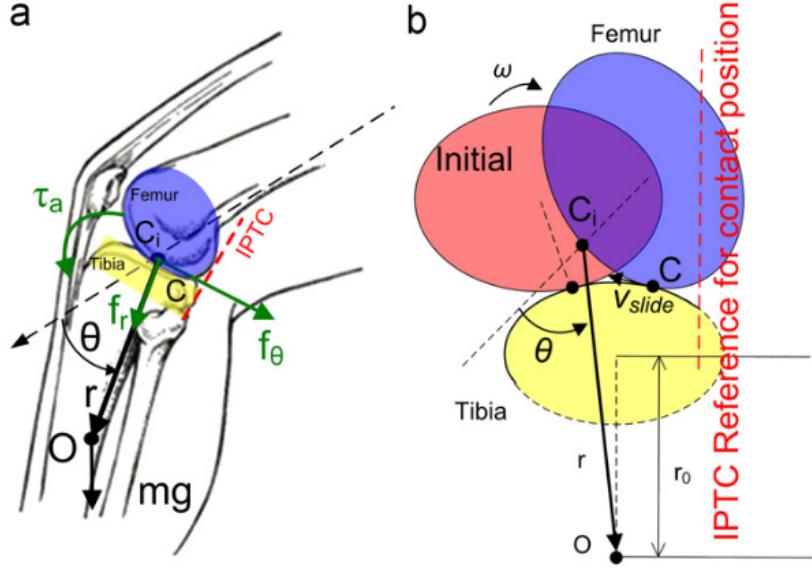


Figure 2.6: Diagram showing knee rotation [16]: (a) demonstrates the tibia's rotation around an initial contact point  $C_i$ . (b) shows the femur and tibia relationship using parameterized shapes between the distance  $r$  and flexion angle  $\theta$

resultant tibia to femur relationship can be seen in Equation 2.1 and Figure 2.7 [16].

$$r(\theta) \text{mm} = 1.078\theta^4 - 11.184\theta^3 + 26.524\theta^2 - 0.825\theta + 263.59 \quad (2.1)$$

These studies measure the relationship of the bones in the knee joints, and not the skin movement around the joint. However, exoskeleton orthosis are usually connected directly to the skin. In order for the research presented above to be applicable to orthosis, a relationship between the skin and femur/tibia must be made. A study by Benoit *et. al* looked at 8 healthy males to compare the bone and

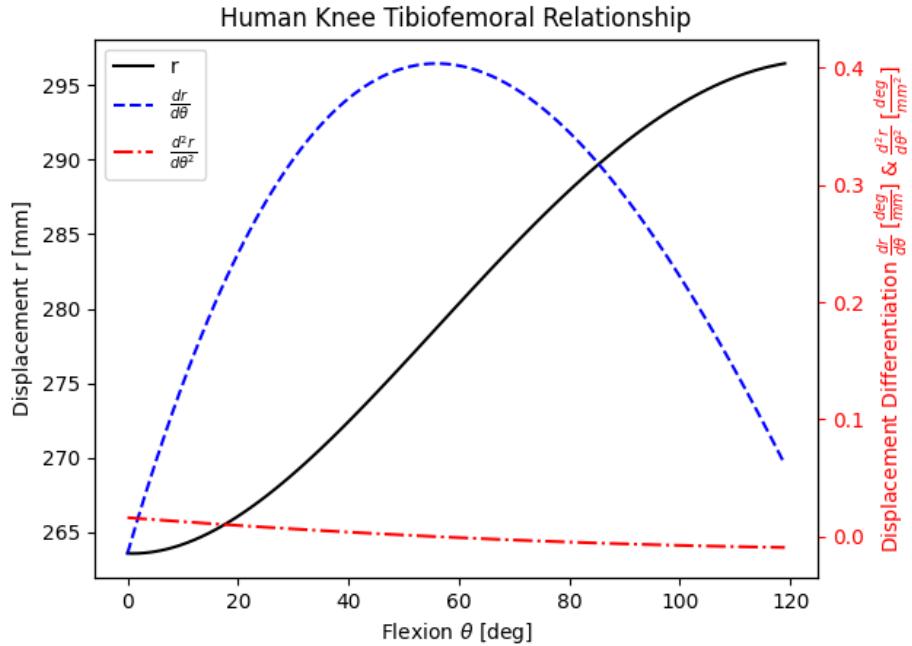


Figure 2.7: Relationship between tibia and femur during the flexion of the joint.  $r(m)$  is the distance between joint point of contact ( $C_i$  in Figure 2.6) and center of mass of the tibia.

skin movement in and around the knee joint in order to identify if skin markers are sufficient to determine bone kinematics around the knee. Each subject had intra-cortical bone-pins inserted in the proximal tibia and distal femur with 3 motion capture markers on each pin. Then, 8 total motion capture markers were attached directly to the skin to measure the difference in the movement (see Figure 2.8).

The results of this study seemed to show a significant difference between the skin and bone movement during several different gaits; average rotational errors were between  $4.4^\circ$  and  $13.1^\circ$  while translational errors averaged up to 16.1mm [1]. What is more interesting is that the errors measured remained relatively constant

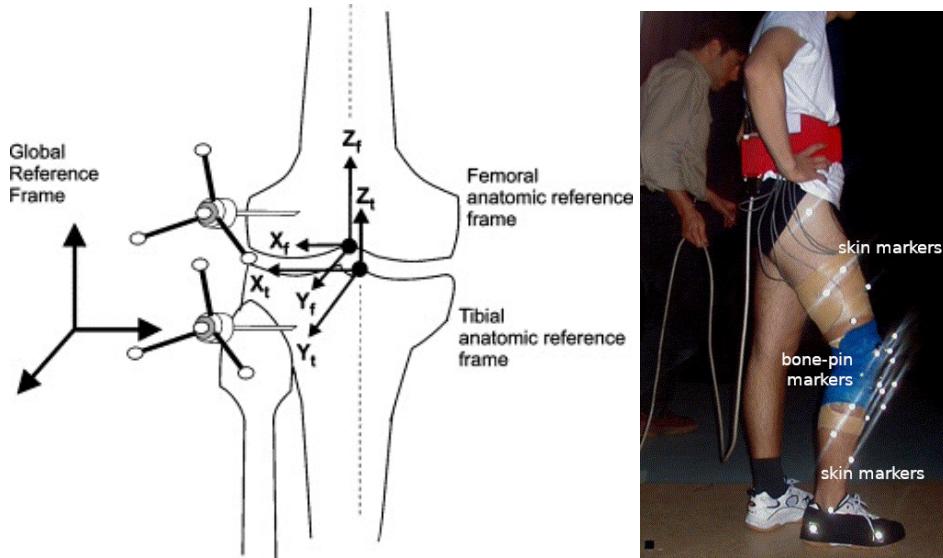


Figure 2.8: Depiction of the research showing the relationship between skin and bone movement: (left) a figure illustrating the location of the bone-pins and (right) showing an image of a subject with all markers attached to them [1]

between different movements, seemingly demonstrating that the error comes from the different connection methods (skin vs bone) and not via measurement tolerances. The researchers were able to conclude that skin-marker kinematics around the knee are not representative of the motion of the bone inside the knee joint. We may be able to therefore inversely conclude that bone kinematics may not be a good representation when designing knee orthoses. However, the research had a relatively small sample size due to the invasive nature of the experiment, and attached markers directly to taped up thighs (see Figure 2.8), which may have failed to represent the actual skin movement. Therefore, more research is needed to conclude the relationship between a skin connection on the thigh and a skin connection on the shank (calf) for the purpose of rehabilitation exoskeletons.

## 2.3 Exoskeleton Orthosis

Exoskeletons are an interesting application to help those with paralysis rehabilitate and exercise their muscles. They can take programmatically reduce or add bodyweight to the user to aid with safe gait training and muscle development. Additionally, powered exoskeletons can help move patient legs in the motion of a gait to help with relearning gait cycles such as walking and climbing stairs. The flexibility these rehabilitation devices offer have been noticed, and several different solutions exist at different levels of clinical implementation.

### 2.3.1 WPI LARRE



Figure 2.9:

The WPI LARRE (Legged Articulated Robotic Rehabilitation Exoskeleton) is an exoskeleton project developed by the Worcester Polytechnic Institute (WPI)

Automated and Interventional Medicine (AiM) Lab. The research presented in this thesis directly contributes to the AiM Lab’s effort to develop an exoskeleton for rehabilitation of lower-limb paralysis patients.

The project was originally started and named as the HEX Gen-1. It’s original goal was to help with rehabilitation of patients who have suffered from spinal chord injury, and took a hybrid approach to an exoskeleton design. The hip joint is powered by a brushless DC motor (Maxon EC 90), while the knee and ankle joints aren’t actively powered by any motor. Instead, the knee joint contains a spring wrap clutch/brake to help provide support to patients throughout their gait cycles. Additionally, the ankle joint included a spring to add force during dorsiflexion to help in walking gaits [21].

### 2.3.2 H2

The H2 robotic exoskeleton is designed by the University of Houston to help stroke survivors with gait training and physical rehabilitation. It has 6 powerful DC motors (3 on each leg on the hip, knee, and ankle joint) geared down with harmonic gearboxes. Each motor is powered by its own local motor controller, with all electronics connected to a main controller via CAN bus. An assistive gait controller is used to apply torque when patients deviate from a planned gait pattern. The device itself was tested on 3 hemiparetic stroke patients, and was safe and effective throughout the 4 week testing period. The pilot clinical study demonstrated the ”assist-as-needed” control system was able to benefit the stroke patients, and help them recover [4].



Figure 2.10: The H2 Exoskeleton, with 6 powered joints and lithium polymer batteries [4]

### 2.3.3 ReWalk

### 2.3.4 EKSO

### 2.3.5 Indigo

### 2.3.6 KINESIS

### 2.3.7 HAL

# **Chapter 3**

## **Knee Joint Design**

### **3.1 Design Requirements**

The following are the design parameters layed out at the beginning of the project:

#### **Follows the defined knee tibiofemoral trajectory**

The knee joint must be able to follow a tibiofemoral trajectory. As referenced in [16], human knee joints can be generally defined by a quartic trajectory. This project will use the parameters of a cadaver, which can be seen in Equation 3.1. The design of the joint must be easily customizable for each patient/user. Ideally, all parts except a few should remain the same to increase simplicity and reduce

cost of manufacturing.

$$r(\theta)mm = 1.078\theta^4 - 11.184\theta^3 + 26.524\theta^2 - 0.825\theta + 263.59 \quad (3.1)$$

### **Supports the mass of a person**

Each joint should be able to support half of the mass of a 85kg human plus a 15kg exoskeleton (total of 100kg) with an additional safety factor.

### **Power/Torque-Speed for walking gaits and sit/stand exercises**

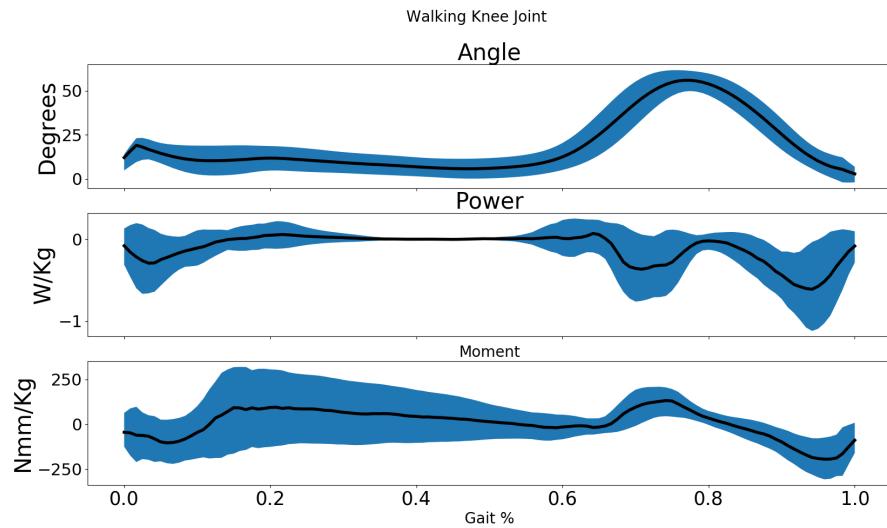


Figure 3.1: Joint kinematics and dynamics during a walking gait cycle [21]

The knee joint has two rehabilitation requirements to fulfill: walking gaits and sit/stand gaits. Prior research has shown that walking gaits require roughly up to

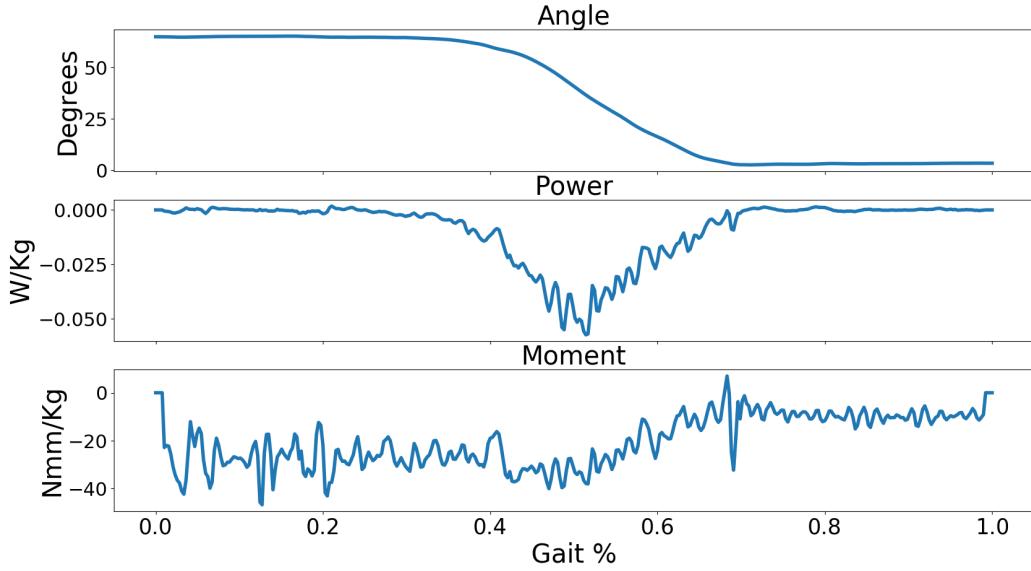


Figure 3.2: Joint kinematics and dynamics during a sit/stand gait cycle [21]

$0.65 \frac{W}{kg}$  and  $0.25 \frac{Nm}{kg}$ , (see Figure 3.1), while a sit/stand gait requires roughly up to  $0.5 \frac{W}{kg}$  and  $0.04 \frac{Nm}{kg}$ . Speed requirements are roughly  $12^\circ/sec$  for walking gaits and  $15^\circ/sec$  for sit/stand gaits. Therefore, the designed knee joint for the  $100kg$  weight specification should be capable of mechanically outputting  $65W$  and  $25Nm$  at  $15^\circ/sec$ .

### Senses the joint angle

Sensors must be able to accurately encode the rotational position of the joint. The rationale behind this requirement is for research, debugging, and most importantly accurate and safe position control. Therefore, the joint must be able to read its own position in both passive (non-powered) modes and active (powered)

modes. It should also have a minimum accuracy of  $\pm 0.5^\circ$  during position control, and be able to maintain position measurement through power cycles (absolute positioning).

### Simple to manufacture and assemble

The joint must be designed with manufacturing and assembly in mind. All components must be easily sourced and generally available. Any machining requirement must be achievable with common machining techniques.

**IMPROVE:**  
This parameter might need to be more measurable?

### Integrates into the WPI LARRE

This research supports the advancement of the WPI LARRE project introduced in subsection 2.3.1. Therefore, the designed joint must be able to integrate into the universal exoskeleton joint connector developed in the LARRE project.

## 3.2 Mechanical Design

The orthotic joint design proposed uses a similar idea to how a human knee joint works; a cam mechanism extends the shank link as it is rotated relative to the thigh link. The joint therefore has two degrees of freedom: rotation around the center of rotation (output shaft of the motor and gearbox) and translation in the direction of the shank. However, since there is only one actuator, the joint is underactuated; this underactuation can be taken advantage of to match a patient's knee trajectory, where the center of mass of the shank extends away from the joint

center as the joint bends. For ease of assembly, the entire joint is held together by 4 M5 shoulder bolts, which also act as the axles for a total of 10 bearings.

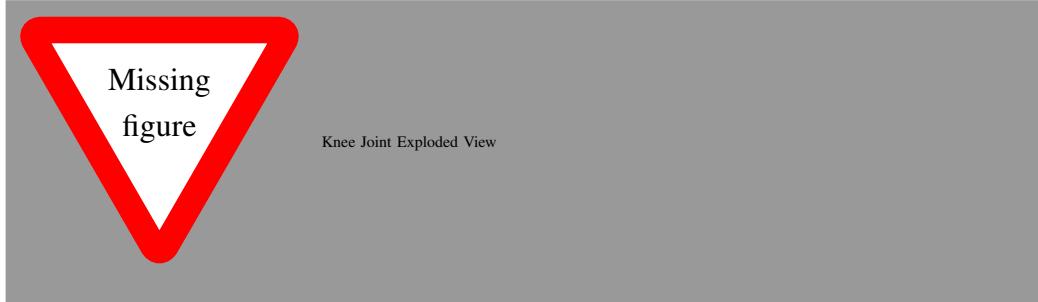


Figure 3.3: Exploded view of the knee joint, with all relevant components labeled

## Torsion Bars

The center of rotation of the joint is designed to match the axis of rotation of the actuator. The output of this actuator is directly connected to the torsion bar using M5 shoulder bolts. Each bolt is designed to support 3 bearings: 2 on the motor side and 1 on the patient side. The reduced count on the patient side allows for the torsion bar to be partially recessed in the shank link to reduce the distance between the center of mass between the patient and the joint. The 6 bearings are still able to support the forces necessary throughout a walking gait cycle (see section 3.2).

**FIX:** Add reference to actuator decision section

## Shank Links

The 2 shank links attach to the lower part of the exoskeleton, and are responsible for taking the rotational energy created by the motor and partially changing it

to translational energy to help linearly extend the shank. The bearings connected to the torsion bars ride in a guide built into the shank link. This guide is slightly larger than the bearing diameter ( 0.3mm) to prevent rubbing without creating much of a backlash (0.39° backlash, see calculation on Equation 3.2).

$$\text{Backlash} = \text{atan}\left(\frac{\frac{0.3\text{mm}}{2}}{22\text{mm}}\right) = 0.39^\circ \quad (3.2)$$

The surface of the guide must be smooth and parallel to the axis of the bearings to avoid damaging them. Depending on the material and manufacturing method chosen, the surface may require additional machining to ensure it can match these requirements. The length of the guide must be larger than the distance between the centers of the two shoulder bolts plus the maximum distance of linear extension by the knee (Equation 3.3). For this prototype, this length was 78mm.

$$\text{GuideLength} \geq \text{TorsionBarC2C} + \text{MaxKneeExtension} = 44\text{mm} + 34\text{mm} = 78\text{mm} \quad (3.3)$$

The shank link is also responsible to connect to the lower part of the exoskeleton. Just like the thigh link, this is done through the universal exoskeleton connector developed throughout the WPI LARRE project [21].

The connection between the thigh link and the shank link is very important, as it adds torsional stability and overall rigidness to the entire joint. It was therefore imperative during the design process to create wide surface contact between the thigh and shank links. To reduce the energy lost to friction between these plates,

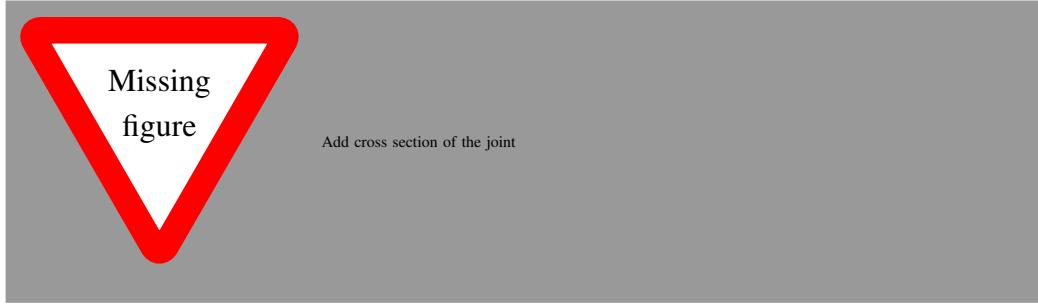


Figure 3.4: A cross section of the knee joint in a 0° position

3.2mm thick Delrin® slides were laser cut and attached to the shank link.

Similarly to the torsion bar, the shank link also uses 2 shoulder bolts to clamp the two shank links on the thigh link as well as to give the bearings that ride on the knee path guide a precise surface to mount to. To maintain a consistent clamping force, lock nuts are used since they do not easily back out with movement and vibration.

### **Thigh Link**

The thigh link acts as the main mounting point for most things, as well as contains the knee path guide. Just like the shank link, the thigh link has the universal exoskeleton connector used throughout the WPI LARRE project. The motor bracket is connected to the thigh link at two locations using  $20mm \times 50mm$  spacers. These spacers must be strong and stiff, as they transmit the torque between the thigh and shank connector in high load situations. A potentiometer is also mounted inside the thigh link to measure the current angle of the joint, as shown in Figure 3.4. The wire connecting to it is routed through a slot in the thigh link to

avoid any interference with the moving shank links. This wire comes out the top and is connected to the main controller of the exoskeleton.

## Knee Path Guide

The knee path guide is built into the thigh link as a slot. The geometry is calculated using several point measurements connected in SolidWorks with a spline. Each point is split by 15 degrees, and calculated from a pre-determined equation. This equation can be measured from a patient knee (see chapter 4), but throughout the design and testing of this knee joint, Equation 3.1 from [16] is used. Figure 3.5 shows the equation above overlayed on the thigh link.

TODO: Verify this number

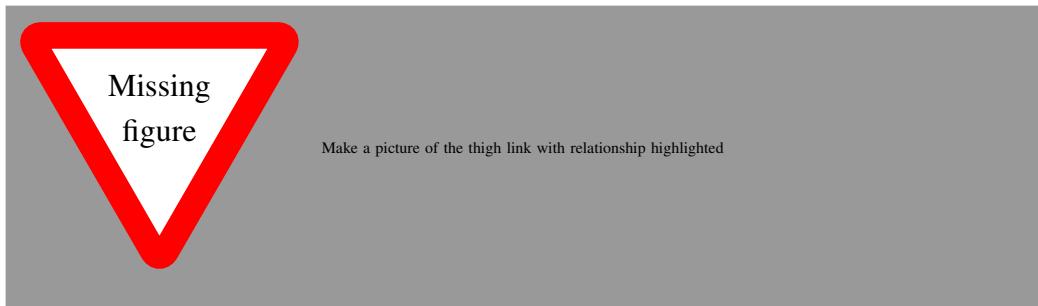


Figure 3.5: The thigh link contains the geometry which the bearings ride on to mimic the tibiofemoral relationship

The joint is designed to be easily adaptable between patients. Therefore, the only customized part in the entire system is the thigh link which holds the knee path guide. All other parts remain the same to decrease cost and improve repairability.

|                                |                     |         |
|--------------------------------|---------------------|---------|
| Input (Motor) Power            | $P_{input}$         | 90Watts |
| Input (Motor) Torque @ Nominal | $\tau_{input}$      | 0.560Nm |
| Input (Motor) Speed @ Nominal  | $\omega_{input}$    | 2510rpm |
| Input (Motor) Stall Torque     | $\tau_{in\_stall}$  | 7.480Nm |
| Gearbox Ration                 | $\frac{n_1}{n_2}$   | 100 : 1 |
| Output Power                   | $P_{output}$        | 81Watts |
| Output Torque @ Nominal        | $\tau_{input}$      | 50.4Nm  |
| Output Speed @ Nominal         | $\omega_{input}$    | 15°/sec |
| Output Stall Torque            | $\tau_{out\_stall}$ | 673.2Nm |

Table 3.1: Motor/gearbox specifications and output power specifications of the proposed joint. See Appendix A for all equations and calculations used.

### Torque Requirements & Actuator Selection

The design parameters specified in section 3.1 are an output of at least 65W and 25Nm at 15°/sec. The Maxon EC90 was chosen, with a peak power output of 90W and a max continuous torque of 0.560Nm at 2510rpm (see Appendix B). To match the speed and torque requirements, a 100 : 1 gearbox ratio is needed. Due to its high reduction to size ratio, a strain wave gearbox from Harmonic Drives™ was chosen.<sup>1</sup> Estimated efficiency of this gearbox is roughly  $\varepsilon = 90\%$ .

The output power of the joint is 81W, with a nominal torque of 50.4Nm at 15°/sec. Power, torque, and speed specifications of the joint theoretically exceed the requirements. Physical testing is needed, however, to ensure that these numbers are accurate and sufficient for a rehabilitation exoskeleton.

---

<sup>1</sup>The gearbox used is proprietary, and no datasheet is available

## Potentiometer and Rotary Encoder

A potentiometer was embedded into the knee design to act as an absolute rotary encoder to measure the current angle of the joint. Its purpose is twofold: to provide for an absolute angle at any given time and to provide for rough rotary encoding when a the motor (for passive experimentation). As mentioned above, the integration needed to protect the sensitive connection points. The potentiometer chosen was the Vishay PRV6, with  $200^\circ$  of travel, a linear resistance, and  $\pm 1\%$  tolerance, which equates to a sensed tolerance of  $\pm 2^\circ$ .

The motor used also has 3 hall sensors used for pinpointing the position of the rotor versus the stator. Since the motor has a 12 poles and 3 sensors (totaling 36 pulses per revolution) as well as a 100:1 reduction through the gearbox, the hall effect sensors can be used to create an effective 3600 pulses per revolution encoder. When used in conjunction with the absolute encoder, the encoded angle can be very precise.

## Bearings

All 10 bearings used in the design are the same (for simplicity and reduction of cost): 19mm outside diameter x 6mm inside diameter x 6mm thick double shielded ball bearings (Model 626ZZ). Each is rated for  $2.6kN$  dynamic load and  $1.05kN$  static load. Before selecting these bearings, two calculations were required to ensure these bearings could support the forces required.

The first is the requirement of the torsion bar. Given the max torque require-

ment for the project is  $25Nm$  and the torsion bar is  $44mm$  from center to center, Equation 3.4 calculates that the total load on all 6 bearings used is  $1136N$ , equaling to roughly  $190N$  per bearing.

$$\text{Total Load per Torsion Bar Bearing : } \frac{1}{6} \times \frac{25Nm}{44mm/2} = \frac{1}{6} \times \frac{25Nm}{0.022m} = 189.4N \quad (3.4)$$

The second force requirement for these bearings were in the knee path cam. Each knee joint must be able to hold half of the weight requirement of  $100kg$  statically. Equation 3.5 demonstrates that each of the 4 bearings used in the cam will see a maximum static load of  $245N$  per bearing.

$$\text{Total Load per Cam Bearing : } \frac{1}{4} \times 100kg \times 9.81m/s = 245.3N \quad (3.5)$$

### 3.3 Material Selection & Manufacturing

**FIX:** Talks only about one component, but never mentions that component. This section should mention every component and the analysis for each one

The concept behind the joint is not dependent on material choice. However, when it came time to manufacture the prototypes, two materials were selected as potential options: aluminum and polylactic acid (PLA) plastic. Aluminum benefits from its strength to weight ratio and manufacturing simplicity when it is being machined. PLA plastic, on the other hand, can be injection molded or 3D

| Material                      | Aluminum | PLA   |
|-------------------------------|----------|-------|
| Mass Density [ $kg/m^3$ ]     | 2700     | 1420  |
| Tensile Strength [ $N/mm^2$ ] | 124.08   | 57.3  |
| Yield Strength [ $N/mm^2$ ]   | 55.15    | 14.3  |
| Shear Modulus [ $N/mm^2$ ]    | 26000    | 55000 |

Table 3.2: Material properties used when analyzing each material in FEA simulation in SolidWorks

printed using fused deposition modeling (FDM) printers. This makes PLA more flexible and less expensive, at the cost of softness and strength when compared to aluminum.

Other plastics and metals were initially considered. Out of the 3D printable plastics that were accessible with the tools available, PLA is strongest, stiffest, and hardest. Other FDM 3D printable plastics considered were acrylonitrile butadiene styrene (ABS) and polyethylene terephthalate (PET). On the metals, side, steels were considered as a material option. However, its density and higher complexity to machine when compared to aluminum ruled it out as a material option.

## Material Analysis

To decide between aluminum and PLA, the materials were analyzed in finite element analysis (FEA) simulation inside Dassault SolidWorks. Table 3.2 shows the material properties used. It is important to note that manufacturing methods were not considered in the analysis; therefore, layer adhesion was not considered when calculating the strength of the material.

explain figures

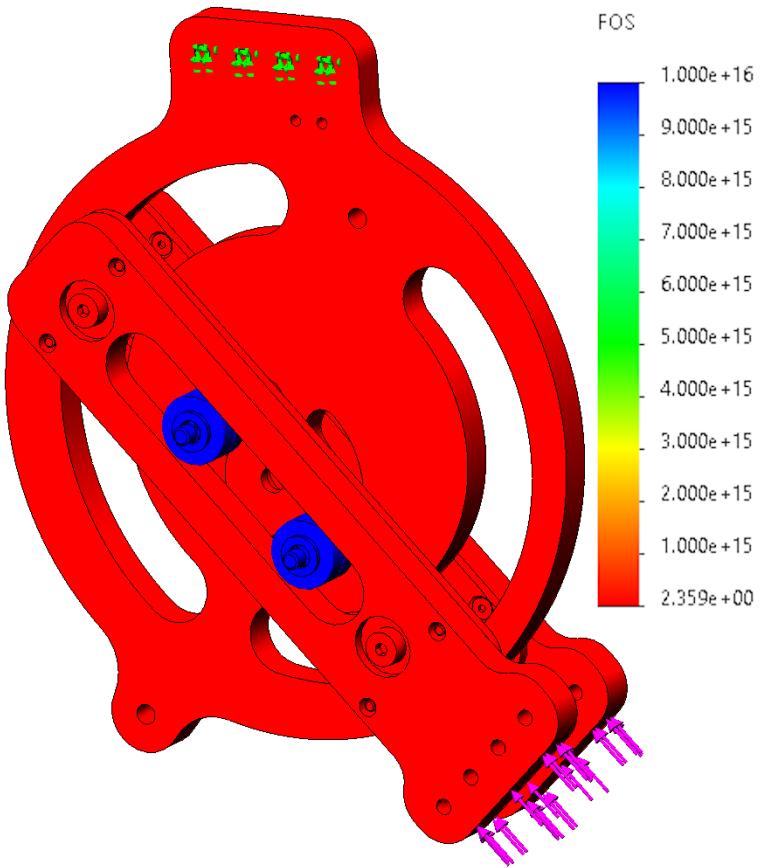


Figure 3.6: FEA of knee joint manufactured from PLA. Force applied (at arrows) is 500N, the resultant safety factor is 2.359.

PLA was chosen as the best material for our experimentation. Analysis demonstrated that it can support the stresses required at angle (shown in Figure 3.8). It can also be manufactured very quickly and easily with access to a conventional FDM 3D printer, allowing for quick revisions during the prototyping process. However, if this joint were to be manufactured for use outside of prototype development and clinical trials, I would recommend using aluminum, as the joint

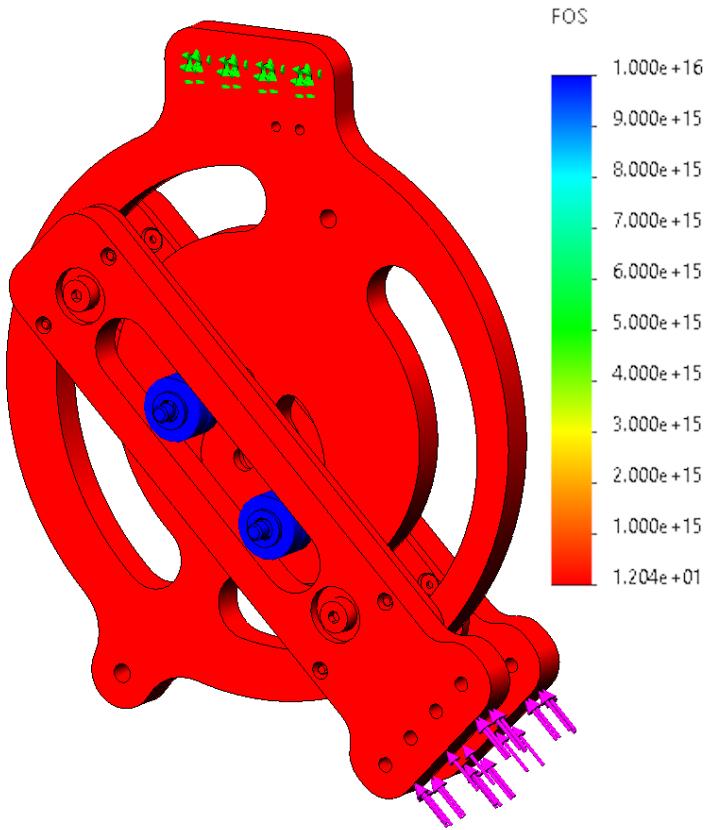


Figure 3.7: FEA of knee joint manufactured from Aluminum. Force applied (at arrows) is 500N, the resultant safety factor is 12.04.

would likely be more resilient and last longer. It would also increase torsional stiffness in the joint and reduce the likelihood of the bearings creating divots in the surface of the knee path guide.

## Manufacturing

The manufacturing process between the two materials is very different as well. While there are many different ways of creating parts in either material, the re-

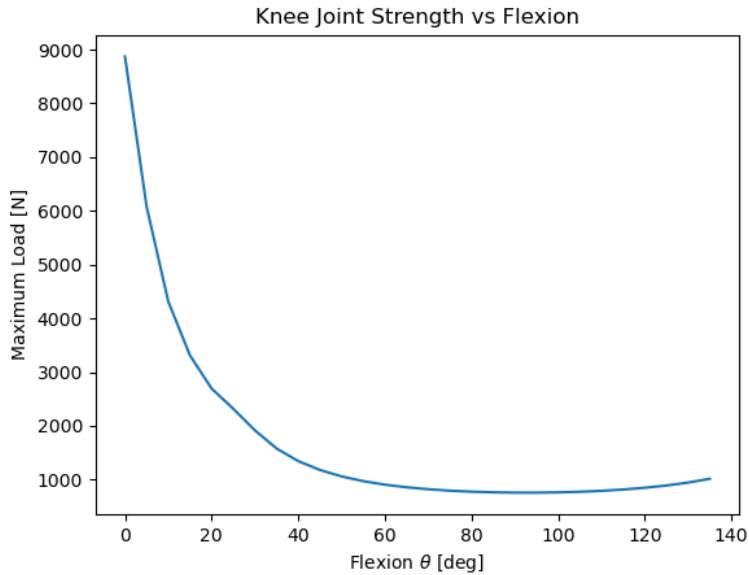


Figure 3.8: Analysis of the joint with major components manufactured out of PLA shows that the design is weakest at a  $90^\circ$  flexion, with a maximum static load of 757N per joint.

search will focus on the most common ways as to be easily replicated by others if desired.

As a plastic, PLA has many options for manufacturing. While PLA can be injection molded or machined, the most common use case for it is through FDM 3D printing. The accessibility and low cost at low production numbers makes this method of manufacturing the best for our use cases for the larger of our parts as well as any part that doesn't deal with big forces. For this project, a Creality Ender 3 was used to 3D print the parts required.

Aluminum can also be 3D printed, but this requires some very specific tools to achieve. It can also be casted (similarly to injection molding for plastics), but

this requires specific machining for the molds, and the parts still usually need to be machined to the final correct dimensions (the best option for high volume manufacturing). Therefore, all aluminum parts were designed to be manufactured using conventional lathes and mills. The manufacturing process, however, can be further simplified with access to a water jet or metal laser cutter. Such a tool can cut out all parts to a rough dimension, and a quick machining pass can finish the surfaces that need to be precise, such as the knee path guide and the slot in the shank link. If water jetting is selected as the preferred method of manufacture, parts may have a slight bevel due to the conical output of the water jet.

### **3.4 Knee Trajectory Testing**

Motion capture and SolidWorks motion simulations were used to verify the joint's trajectory based on an inputted tibiofemoral trajectory. The motion simulation outputted a perfect match to the input equation (Equation 3.1), since the simulation platform is using a perfect model which directly inputs the equation above.

To further verify the relationship, a 10-camera Vicon Vantage 5 motion capture system was used. 9 motion capture dots were placed strategically around the knee joint. Two rigid bodies were used (each containing 4 motion capture dots) to be able to measure position and orientation in all 6 degrees of freedom. One rigid body was placed on the connector on the shank link, while the other was attached to the connector of the thigh link. A final motion capture dot was placed at the joint

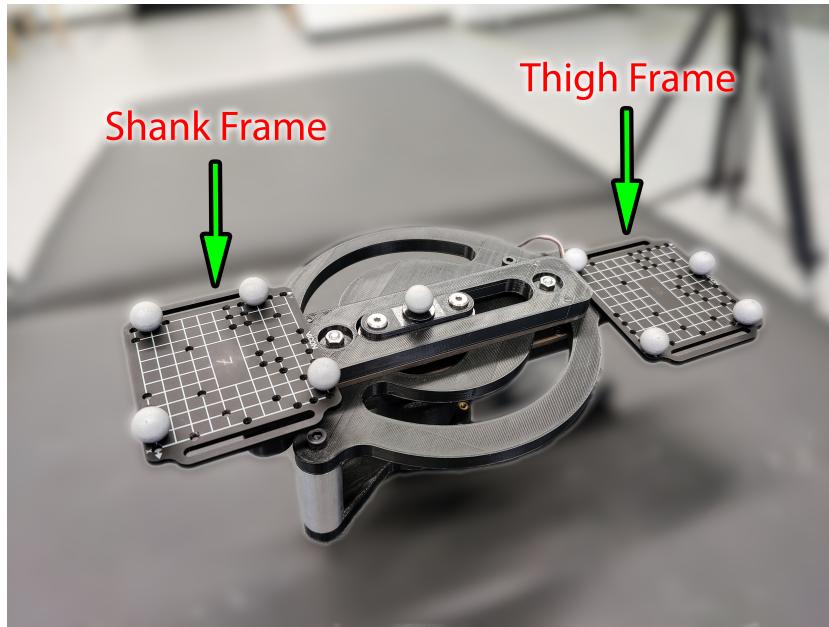


Figure 3.9: Experimental setup for measuring the trajectory of the manufactured joint. 9 motion capture dots were used to measure any movement in all 6 degrees of freedom.

center. Then, the joint was manually actuated through its range while collecting data from the motion capture system. The data was processed using software tools developed in chapter 4 for measuring human tibiofemoral relationships. To ensure the data collected remained representative of the test and was not modified by any processing tools developed, only data importing tools were used. These tools simply took the raw data from the Vicon system and imported it into Python in a cleaner way.

The motion capture analysis data in Figure 3.10 demonstrates the effectiveness of the joint; it was able to follow the desired trajectory layed out in Equation 3.1 with minimal error.

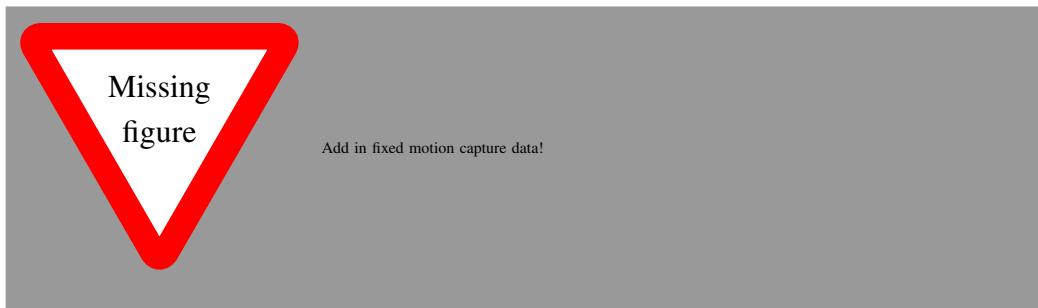


Figure 3.10: Results from the motion capture system demonstrates that the designed knee joint can follow a desired trajectory very closely, only deviating by [insert number here]

**TODO:** More text needed here!

# **Chapter 4**

## **Parameterization of Human Knee Joints**

# **Chapter 5**

## **Conclusion & Future Work**

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# Appendix A

## Joint Power/Torque/Speed Calculations

|                                |                    |         |
|--------------------------------|--------------------|---------|
| Input (Motor) Power            | $P_{input}$        | 90Watts |
| Input (Motor) Torque @ Nominal | $\tau_{input}$     | 0.560Nm |
| Input (Motor) Speed @ Nominal  | $\omega_{input}$   | 2510rpm |
| Input (Motor) Stall Torque     | $\tau_{in\_stall}$ | 7.480Nm |
| Gearbox Ration                 | $\frac{n_1}{n_2}$  | 100 : 1 |

Table A.1: Motor/Gearbox Specifications

### Power Calculation

$$P_{output} = \epsilon P_{input} = 0.9 * 90W = 81W \quad (\text{A.1})$$

## Torque Calculation

$$\tau_{output} = \epsilon \tau_{input} \frac{n_1}{n_2} = 0.9 * 0.560 Nm * \frac{100}{1} = 50.4 Nm \quad (A.2)$$

## Speed Calculation

$$\omega_{output} = \omega_{input} \frac{n_2}{n_1} = 2510 rpm * \frac{100}{1} = 2.510 rpm = 15^\circ/sec \quad (A.3)$$

|                         |                     |         |
|-------------------------|---------------------|---------|
| Output Power            | $P_{output}$        | 81Watts |
| Output Torque @ Nominal | $\tau_{input}$      | 50.4Nm  |
| Output Speed @ Nominal  | $\omega_{input}$    | 15°/sec |
| Output Stall Torque     | $\tau_{out\_stall}$ | 673.2Nm |

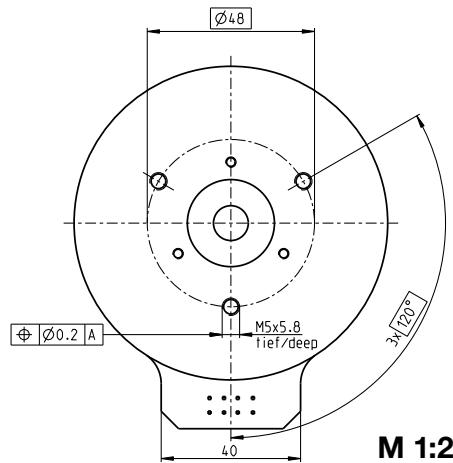
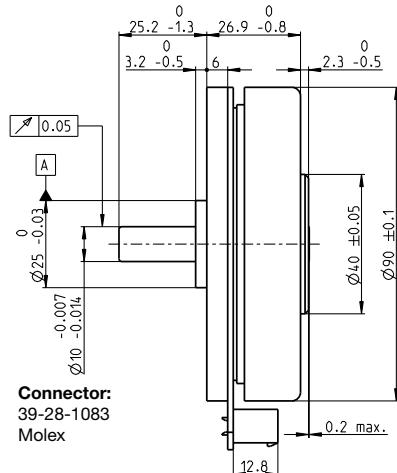
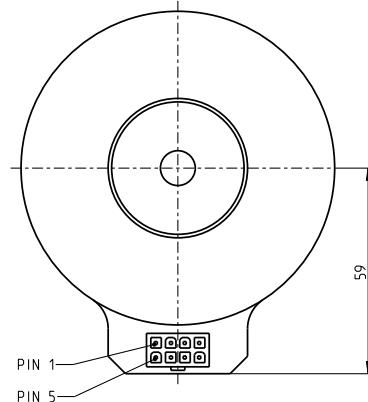
Table A.2: Joint Output Specifications

## **Appendix B**

### **Maxon Motor EC90 Datasheet**

Below is the datasheet for the motor chosen for this project: Maxon EC90 part number 429271 (highlighted)

# EC 90 flat Ø90 mm, brushless, 90 Watt



M 1:2

- Stock program
- Standard program
- Special program (on request)

## Part Numbers

with Hall sensors

**323772**    429271    **244879**

### Motor Data

#### Values at nominal voltage

|   |     |      |      |      |
|---|-----|------|------|------|
| 1 Nominal voltage                           | V   | 24   | 36   | 48   |
| 2 No load speed                             | rpm | 3190 | 3120 | 2080 |
| 3 No load current                           | mA  | 544  | 348  | 135  |
| 4 Nominal speed                             | rpm | 2590 | 2510 | 1610 |
| 5 Nominal torque (max. continuous torque)   | mNm | 444  | 560  | 533  |
| 6 Nominal current (max. continuous current) | A   | 6.06 | 4.76 | 2.27 |
| 7 Stall torque                              | mNm | 4940 | 7480 | 4570 |
| 8 Stall current                             | A   | 70   | 69   | 21.1 |
| 9 Max. efficiency                           | %   | 84   | 87   | 85   |

#### Characteristics

|                                       |         |       |       |       |
|---------------------------------------|---------|-------|-------|-------|
| 10 Terminal resistance phase to phase | Ω       | 0.343 | 0.522 | 2.28  |
| 11 Terminal inductance phase to phase | mH      | 0.264 | 0.625 | 2.5   |
| 12 Torque constant                    | mNm/A   | 70.5  | 109   | 217   |
| 13 Speed constant                     | rpm/V   | 135   | 88    | 44    |
| 14 Speed/torque gradient              | rpm/mNm | 0.659 | 0.423 | 0.462 |
| 15 Mechanical time constant           | ms      | 21.1  | 13.6  | 14.8  |
| 16 Rotor inertia                      | gcm²    | 3060  | 3060  | 3060  |

### Specifications

#### Thermal data

|                                       |              |
|---------------------------------------|--------------|
| 17 Thermal resistance housing-ambient | 1.91 K/W     |
| 18 Thermal resistance winding-housing | 2.6 K/W      |
| 19 Thermal time constant winding      | 46 s         |
| 20 Thermal time constant motor        | 283 s        |
| 21 Ambient temperature                | -40...+100°C |
| 22 Max. winding temperature           | +125°C       |

#### Mechanical data (preloaded ball bearings)

|                                       |           |
|---------------------------------------|-----------|
| 23 Max. speed                         | 5000 rpm  |
| 24 Axial play at axial load < 15 N    | 0 mm      |
| > 15 N                                | 0.14 mm   |
| 25 Radial play                        | preloaded |
| 26 Max. axial load (dynamic)          | 12 N      |
| 27 Max. force for press fits (static) | 183 N     |
| (static, shaft supported)             | 8000 N    |
| 28 Max. radial load, 5 mm from flange | 68 N      |

#### Other specifications

|                         |       |
|-------------------------|-------|
| 29 Number of pole pairs | 12    |
| 30 Number of phases     | 3     |
| 31 Weight of motor      | 600 g |

Values listed in the table are nominal.

#### Connection

|       |                         |
|-------|-------------------------|
| Pin 1 | Hall sensor 1           |
| Pin 2 | Hall sensor 2           |
| Pin 3 | $V_{Hall}$ 4.5...18 VDC |
| Pin 4 | Motor winding 3         |
| Pin 5 | Hall sensor 3           |
| Pin 6 | GND                     |
| Pin 7 | Motor winding 1         |
| Pin 8 | Motor winding 2         |

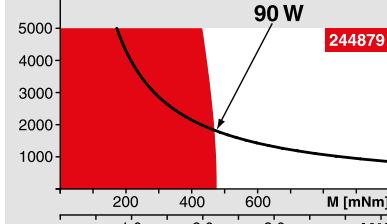
Wiring diagram for Hall sensors see p. 43

#### Cable

Connection cable Universal, L = 500 mm    **339380**  
Connection cable to EPOS2, L = 500 mm    **354045**

### Operating Range

n [rpm]



### Comments

#### Continuous operation

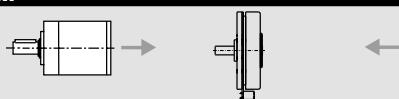
In observation of above listed thermal resistance (lines 17 and 18) the maximum permissible winding temperature will be reached during continuous operation at 25°C ambient.  
= Thermal limit.

#### Short term operation

The motor may be briefly overloaded (recurring).

— Assigned power rating

### maxon Modular System



Overview on page 28-36

#### Planetary Gearhead

Ø52 mm

4 - 30 Nm

Page 351

#### Encoder MILE

512 - 6400 CPT,  
2 channels  
Page 390

#### Recommended Electronics:

|                         |         |
|-------------------------|---------|
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| ESCON Mod. 50/4 EC-S    | 427     |
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