

# EE302 HW1 SOLUTIONS

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Q1- Input:  $y_1$        $M_{\text{body}} \ddot{y}_3 = -k_2(y_3 - y_2) - b(\dot{y}_3 - \dot{y}_2)$   
 a- Output:  $y_3$        $M_{\text{tire}} \ddot{y}_2 = -k_2(y_2 - y_3) - b(\dot{y}_2 - \dot{y}_3) - k_1(y_2 - y_1)$

$$\Rightarrow \underbrace{(M_{\text{body}} s^2 + bs + k_2)}_{\triangleq D_1(s)} y_3(s) = (bs + k_2) y_2(s)$$

$$\underbrace{(M_{\text{tire}} s^2 + bs + k_1 + k_2)}_{\triangleq D_2(s)} y_2(s) = (bs + k_2) y_3(s) + k_1 y_1(s)$$

$$\Rightarrow \begin{aligned} D_1 y_3 &= (bs + k_2) y_2 \Rightarrow D_1 D_2 y_3 = (bs + k_2) ((bs + k_2) y_3 + k_1 y_1) \\ D_2 y_2 &= (bs + k_2) y_3 + k_1 y_1 \end{aligned}$$

$$\Rightarrow (D_1 D_2 - (bs + k_2)^2) y_3 = (bs + k_2) k_1 y_1$$

$$\Rightarrow \frac{y_3(s)}{y_1(s)} = \frac{k_1 (bs + k_2)}{D_1(s) D_2(s) - (bs + k_2)^2}$$

b- System is 4<sup>th</sup> order (order of the denominator polynomial)

$$x \triangleq \begin{bmatrix} y_3 \\ \dot{y}_3 \\ y_2 \\ \dot{y}_2 \end{bmatrix} \quad \dot{x} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ -\frac{k_2}{M_{\text{body}}} & -\frac{b}{M_{\text{body}}} & \frac{k_2}{M_{\text{body}}} & \frac{b}{M_{\text{body}}} \\ 0 & 0 & 0 & 1 \\ \frac{k_2}{M_{\text{tire}}} & \frac{b}{M_{\text{tire}}} & -\frac{k_1 + k_2}{M_{\text{tire}}} & -\frac{b}{M_{\text{tire}}} \end{bmatrix} x + \begin{bmatrix} 0 \\ 0 \\ 0 \\ \frac{k_1}{M_{\text{tire}}} \end{bmatrix} u \quad u = y_1$$

$$y = \begin{bmatrix} 1 & 0 & 0 & 0 \end{bmatrix} x$$

↓  
output  $y_3$



Q2-) a-)

$$v_p = K_\theta (\theta_r - \theta_o)$$

$$v_p = L_p \frac{di_p}{dt} + R_p i_p$$

$$v_g = K_g i_p$$

$$v_g - v_b = \underbrace{(L_g + L_m)}_{L_t} \frac{di_m}{dt} + \underbrace{(R_g + R_m)}_{R_t} i_m$$

$$\dot{\theta}_o = \omega_o$$

$$v_b = K_b \omega_o$$

$$T_o = K_z i_m$$

$$J \dot{\omega}_o = T_o - B \omega_o$$

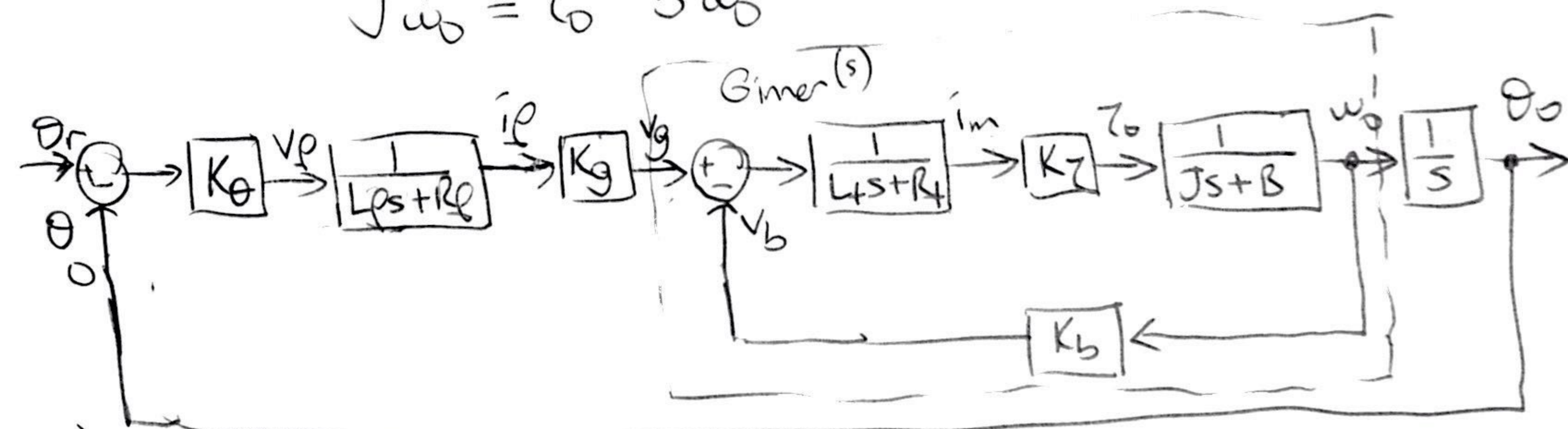
$\theta_r$ : input

$\theta_o$ : output

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$$L_t \triangleq L_g + L_m$$

$$R_t \triangleq R_g + R_m$$



b-)

$$G_{imer}(s) = \frac{K_z}{(L_t s + R_t)(J s + B)} = \frac{K_z}{1 + \frac{K_z K_b}{(L_t s + R_t)(J s + B)}}$$

$$G(s) = \frac{\theta_o(s)}{\theta_r(s)} = \frac{K_\theta K_g K_z}{1 + \frac{K_\theta K_g K_z}{s(L_p s + R_p)((L_t s + R_t)(J s + B) + K_z K_b)}}$$



c-) System is 4<sup>th</sup> order. Hence the minimum # of (3-  
state variables to use is 4.

d-) i. is not possible since  $\theta_r$  is input, it cannot be  
in the state.

ii. is possible.

iii. is not possible since there is no equation to  
express  $\ddot{\theta}_0$  in terms of state variables.

Also the output equation cannot be written  
with this state definition.

iv. is not possible since  $z_0 = K_z i_m$ . Hence  
two state variables are linear combinations of each  
other. Hence the effective number of state  
variables is 3. We need at least 4 state  
variables which cannot be written as a linear  
combination of each other.

As a result, only ii. is suitable for writing  
a state-space representation for this system.

$$\Rightarrow x \triangleq [i_f \ i_m \ \theta_0 \ \dot{\theta}_0]^T$$

d-) (Continued)

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$$L_f \dot{i}_f = K_\theta (\theta_r - \theta_o) - R_f i_f$$

$$L_t \dot{i}_m = \underbrace{K_g i_f}_{v_g} - \underbrace{K_b \omega_o}_{v_b} - R_t i_m$$

$$\dot{\theta}_o = \omega_o$$

$$J \dot{\omega}_o = \underbrace{K_z i_m}_{\tau_o} - B \omega_o$$

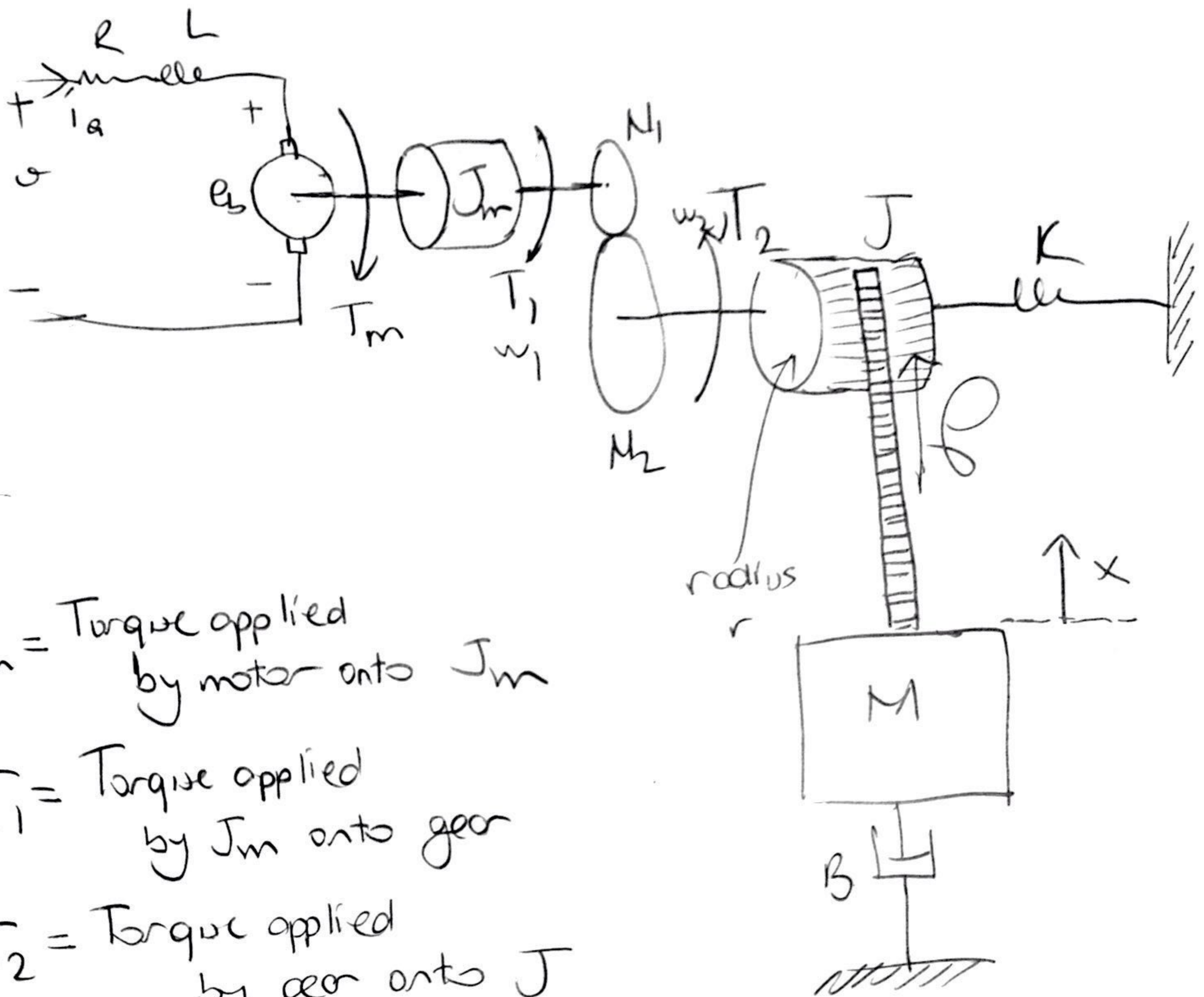
$$\dot{\hat{x}} = \begin{bmatrix} -\frac{R_f}{L_f} & 0 & -\frac{K_\theta}{L_f} & 0 \\ \frac{K_g}{L_t} & -\frac{R_t}{L_t} & 0 & -\frac{K_b}{L_t} \\ 0 & 0 & 0 & 1 \\ 0 & \frac{K_z}{J} & 0 & -\frac{B}{J} \end{bmatrix} x + \begin{bmatrix} \frac{K_\theta}{L_f} \\ 0 \\ 0 \\ 0 \end{bmatrix} \underbrace{u}_{\theta_r}$$

$$y = \begin{bmatrix} 0 & 0 & 1 & 0 \end{bmatrix} x$$



Q3) A more detailed schematic diagram is given below.

(5)



$T_m$  = Torque applied by motor onto  $J_m$

$T_1$  = Torque applied by  $J_m$  onto gear

$T_2$  = Torque applied by gear onto  $J$

$f$ ; Force applied by  $J$  onto ideal gear and  $M$ ,  
a)

$$v - e_b = L \frac{di_a}{dt} + R i_a \quad T_2 = \frac{N_2}{N_1} T_1 \quad \dot{\theta}_2 = \omega_2$$

$$T_m = K_m i_a$$

$$e_b = K_b \omega_1$$

$$\dot{\theta}_1 = \omega_1$$

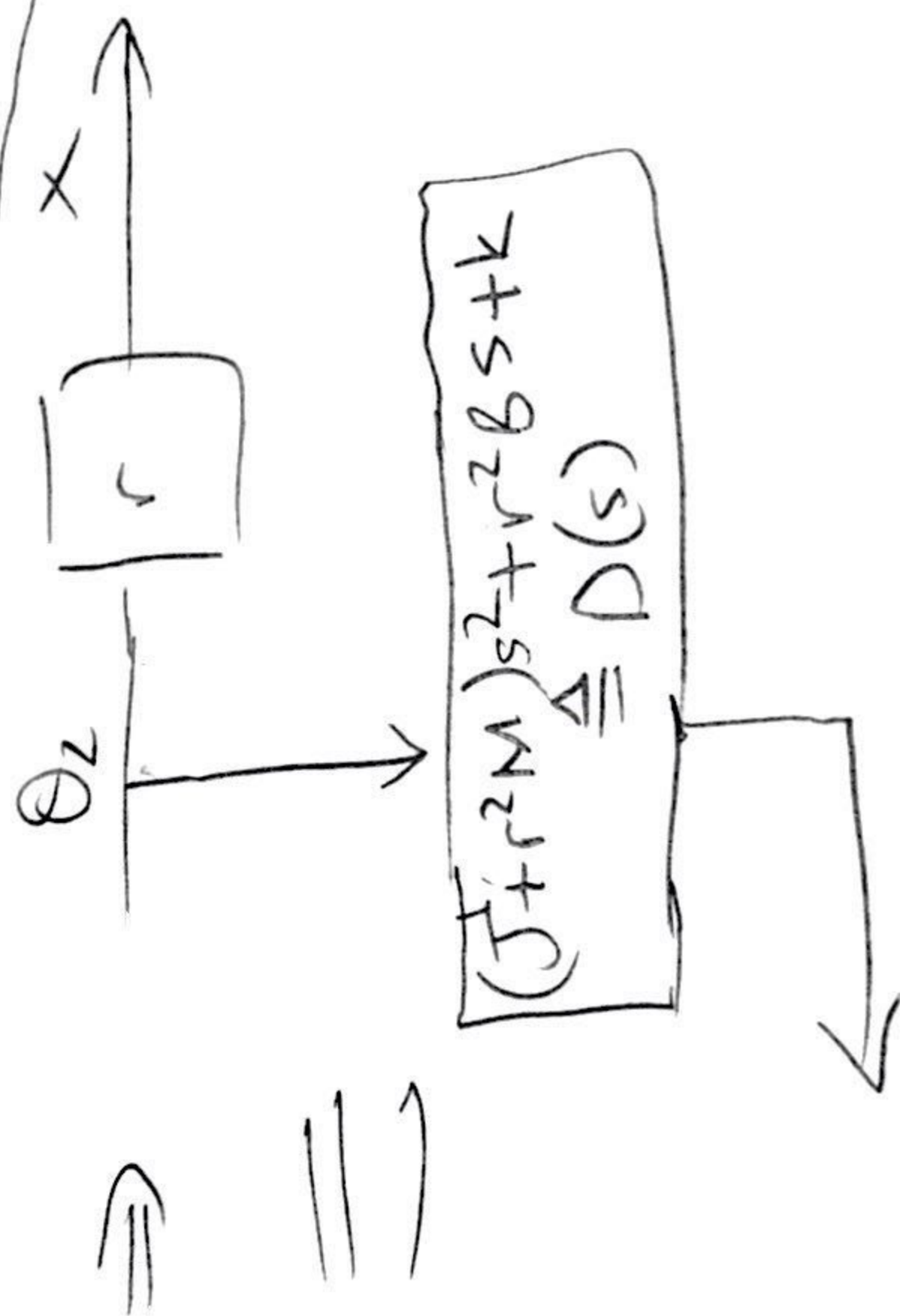
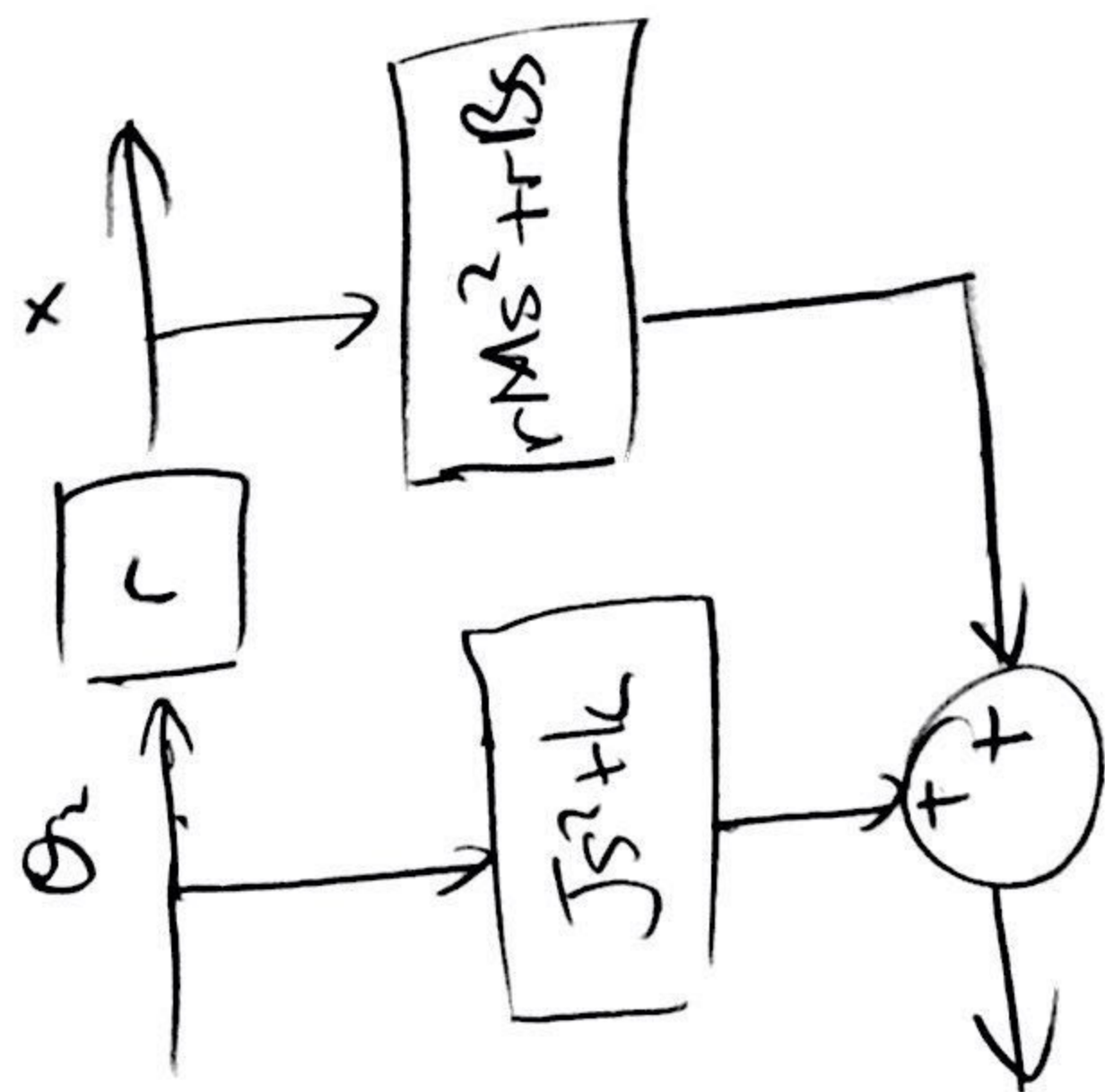
$$J_m \dot{\omega}_1 = T_m - T_1$$

$$J \ddot{\theta}_2 = T_2 - f - k \theta_2$$


$$M \ddot{x} = +f - B \dot{x}$$

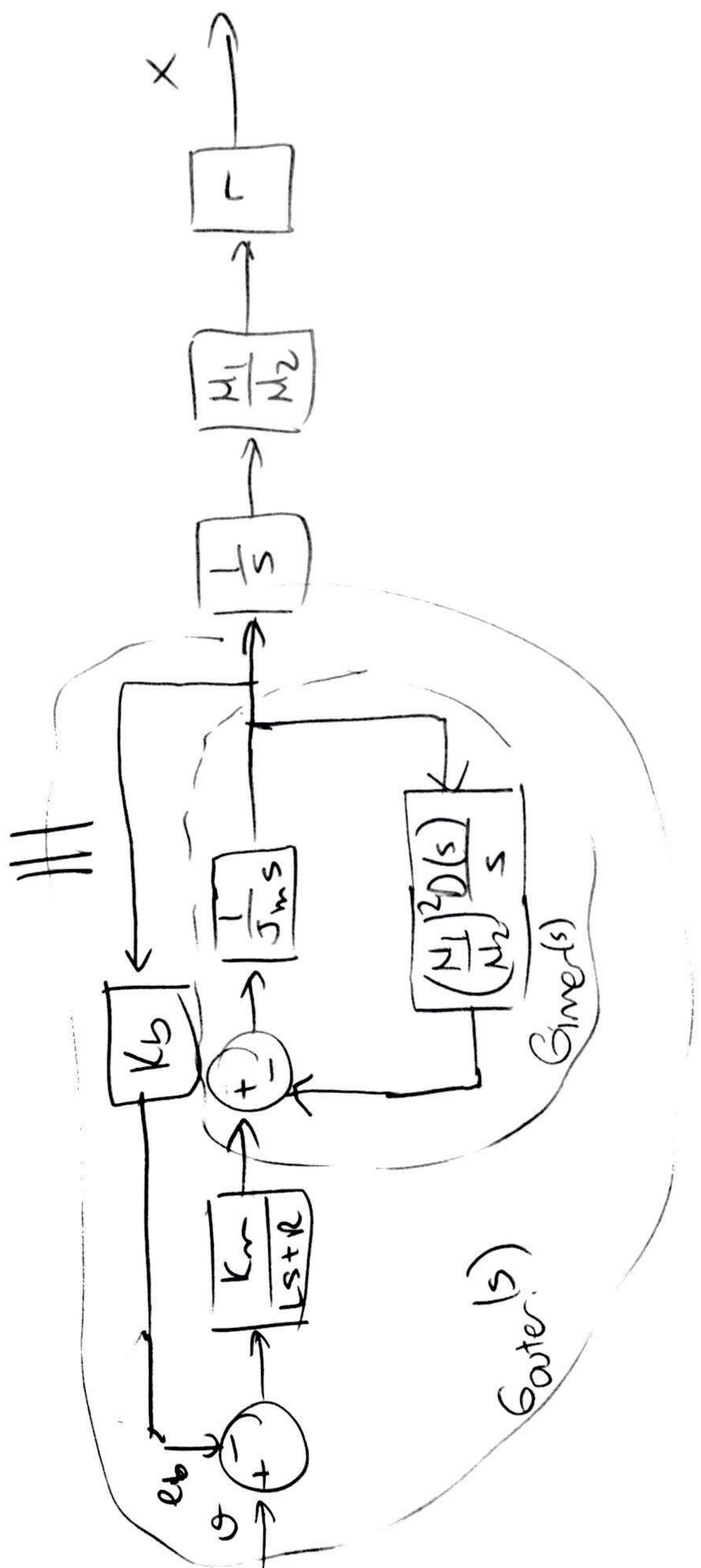
$$x = r \theta_2$$













$$G_{iner}(s) = \frac{\frac{1}{J_m s}}{1 + \left(\frac{M_1}{M_2}\right)^2 \frac{D(s)}{J_m s^2}} = \frac{s}{J_m s^2 + \left(\frac{M_1}{M_2}\right)^2 D(s)}$$

$$G_{outer}(s) = \frac{K_{ms}}{(Ls + R) \left( J_m s^2 + \left(\frac{M_1}{M_2}\right)^2 D(s) \right)} = \frac{K_{ms}}{1 + \frac{K_b K_{ms}}{(Ls + R) \left( J_m s^2 + \left(\frac{M_1}{M_2}\right)^2 D(s) \right)}} + K_b K_{ms}$$

$$G(s) = \frac{X(s)}{V(s)} = \frac{\left(\frac{M_1}{M_2}\right) r K_m}{(Ls + R) \left( J_m s^2 + \left(\frac{M_1}{M_2}\right)^2 D(s) \right) + K_b K_{ms}}$$

$$J_m s^2 + \left(\frac{M_1}{M_2}\right)^2 (J + r^2 M) s^2 + \left(\frac{M_1}{M_2}\right)^2 r^2 B s + \left(\frac{M_1}{M_2}\right)^2 K = J_{eq} s^2 + B_{eq} s + K_{eq}$$

$$\Rightarrow J_{eq} \triangleq J_m + \left(\frac{M_1}{M_2}\right)^2 (J + r^2 M)$$

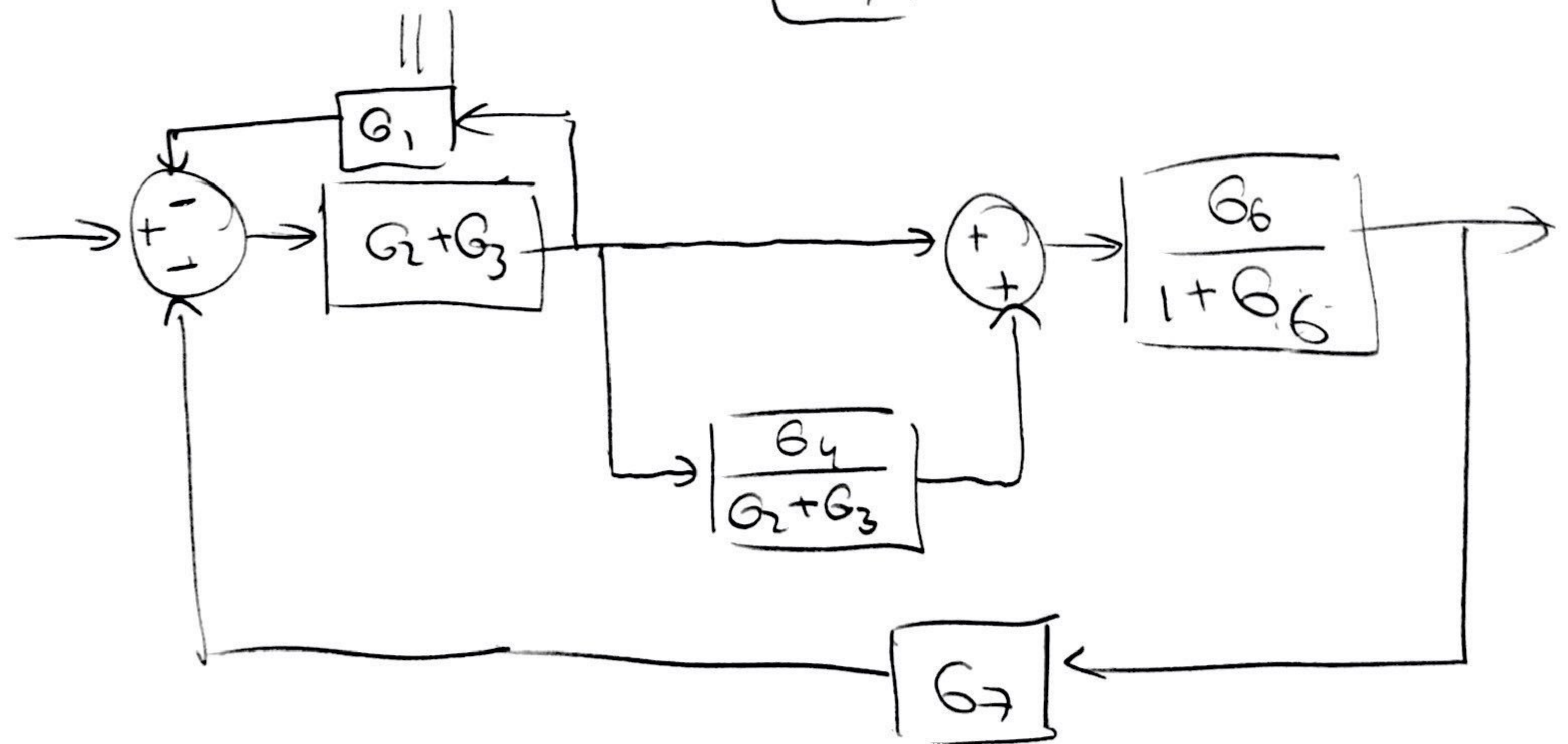
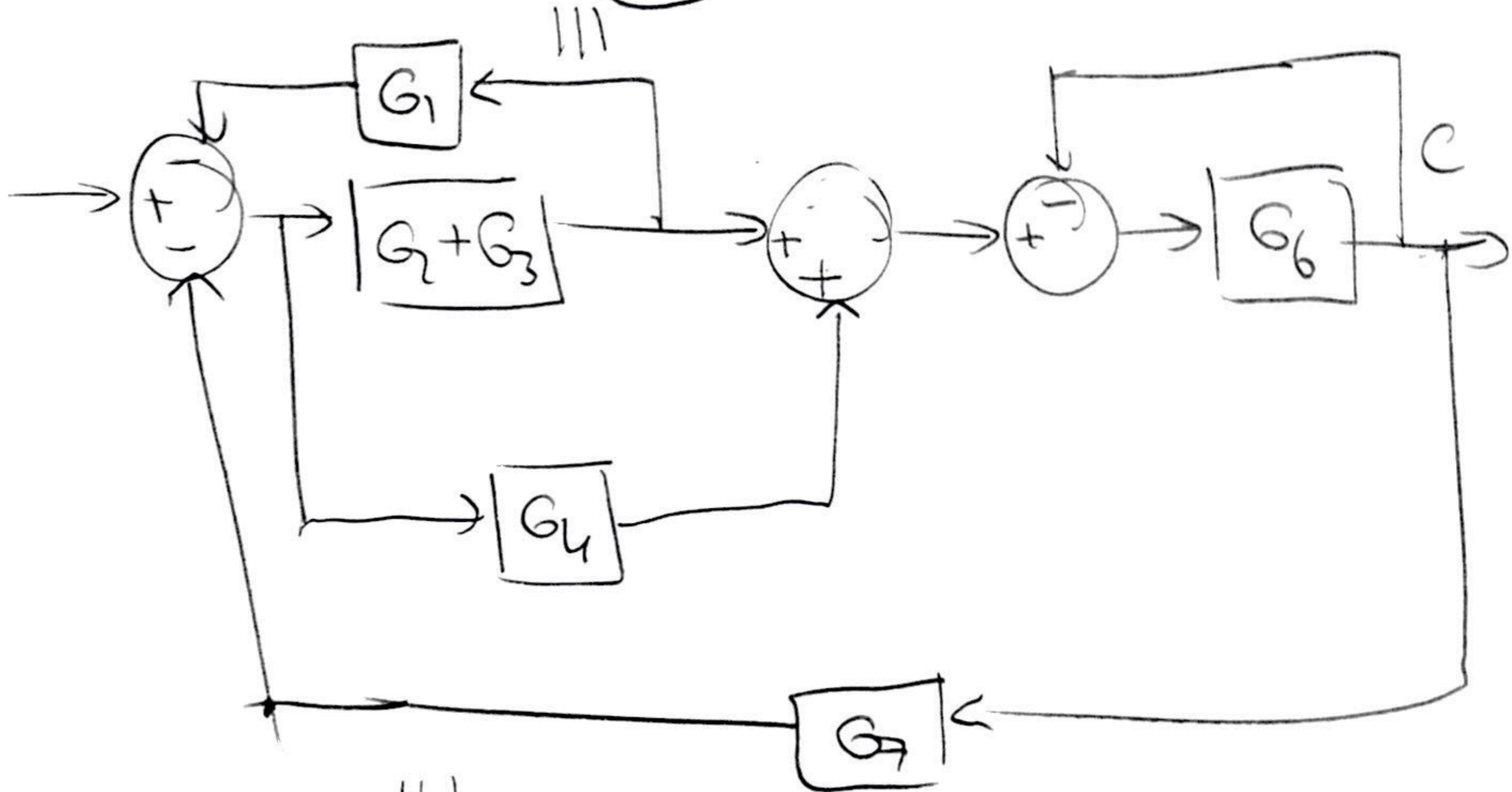
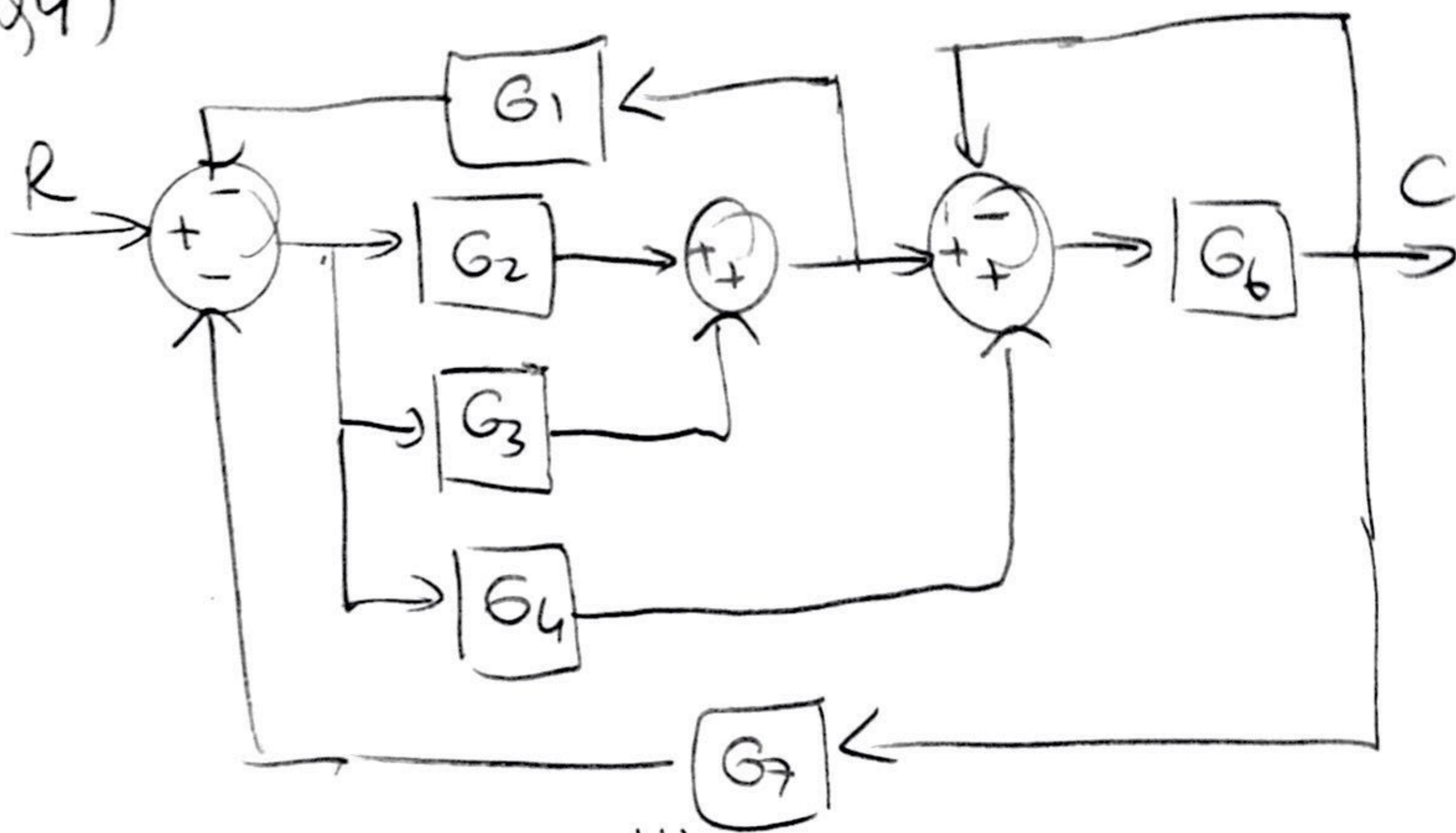
$$B_{eq} = \left(\frac{M_1}{M_2}\right)^2 r^2 B$$

$$K_{eq} = \left(\frac{M_1}{M_2}\right)^2 K$$

$$G(s) = \frac{\left(\frac{M_1}{M_2}\right) r K_m}{(Ls + R) (J_{eq} s^2 + B_{eq} s + K_{eq})}$$



Q4)





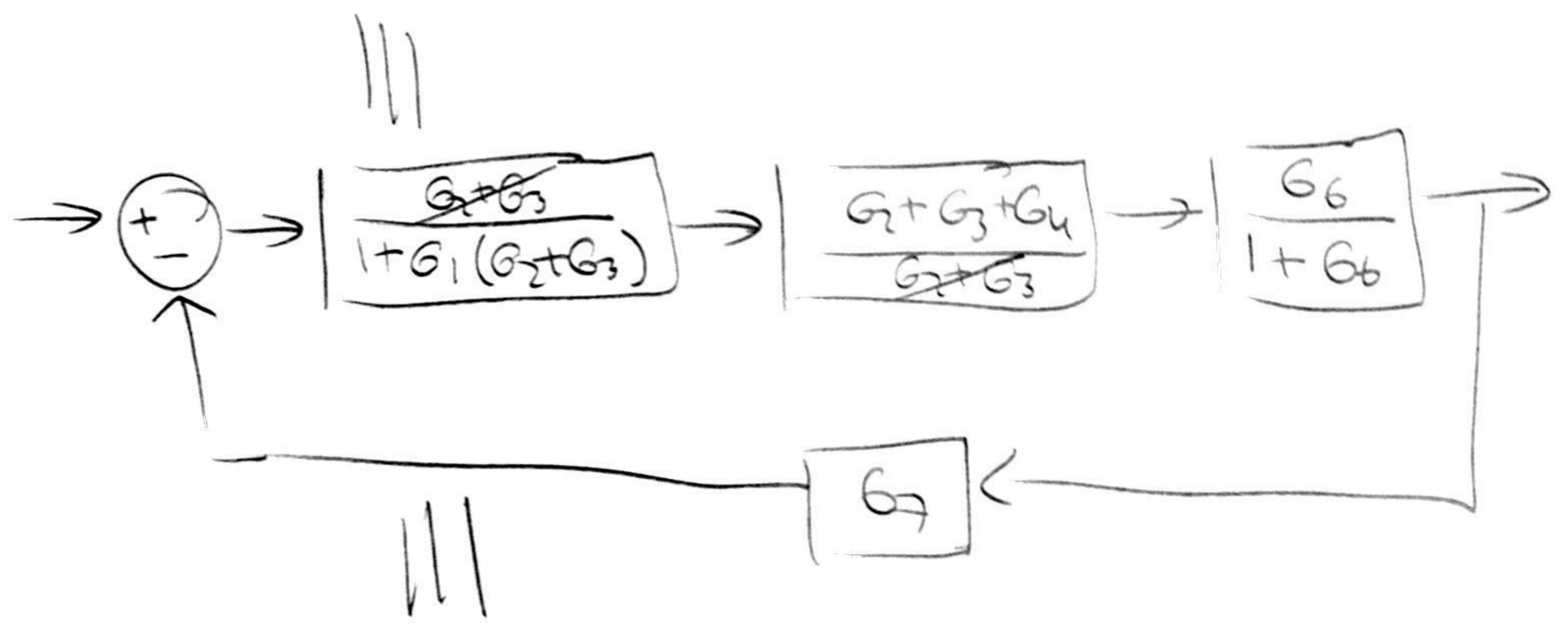
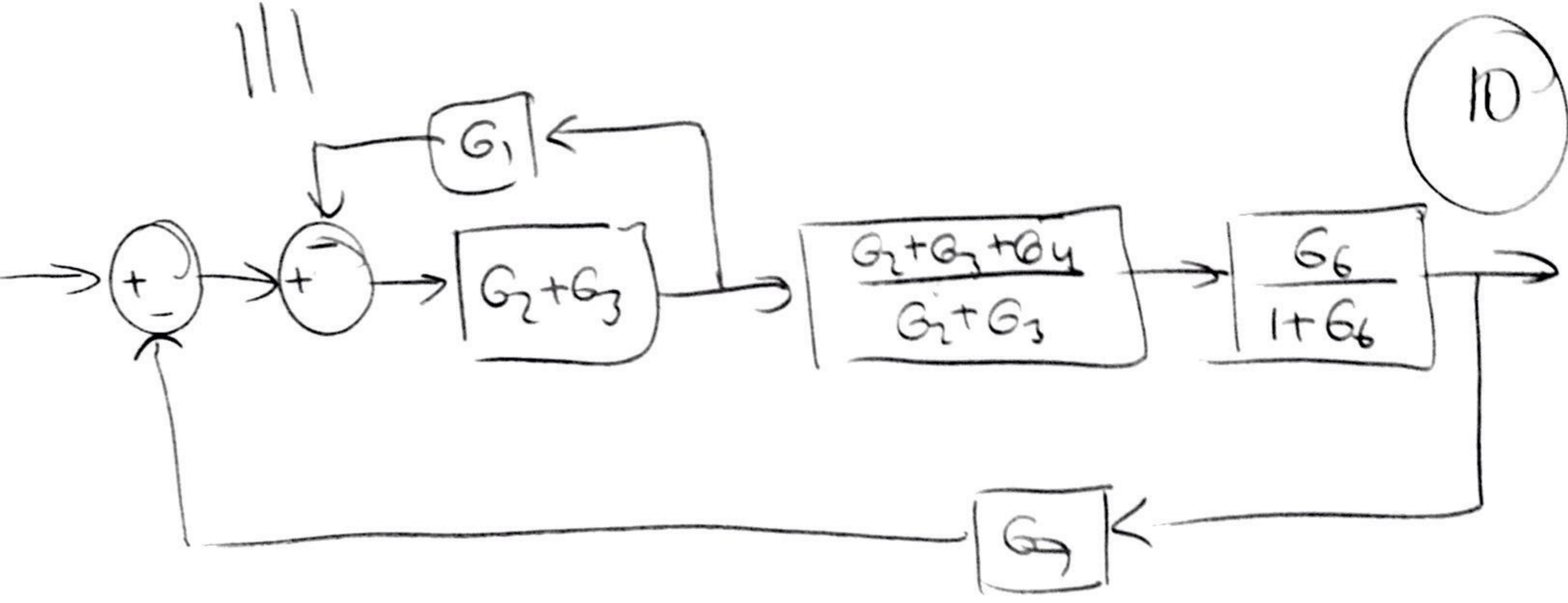


Diagram 3: Simplified transfer function. The input signal enters a block containing the transfer function:

$$1 + \frac{G_6(G_2 + G_3 + G_4)}{(1 + G_6)(1 + G_1(G_2 + G_3))}$$

The output of this block is the system output.

Diagram 4: Final simplified transfer function. The input signal enters a block containing the transfer function:

$$\frac{G_6(G_2 + G_3 + G_4)}{(1 + G_6)(1 + G_1(G_2 + G_3) + G_6 G_7(G_2 + G_3 + G_4))}$$

The output of this block is the system output.