

TTK4150 Nonlinear Control Systems

Lecture 12

Feedback Linearization



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Lecture 12: Feedback Linearization

Introduction Previous lecture

Previous lectures on nonlinear control design



Previous lectures on nonlinear control design:

- Lyapunov based control design
- Cascaded control: Lemma 4.7 allows for modular design
(And background material, Sontag and Loria)
- Passivity-based control design

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Lecture 12: Feedback Linearization

Introduction Today's goals

Outline I



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Lecture 12: Feedback Linearization

Outline II

- Summary: Input-output linearization for stabilization
- Summary: Input-output linearization for tracking control
- Application example
- Advantages/shortcomings

Today's goals

After today you should...

- Know the concepts of relative degree, normal form, external dynamics, internal dynamics and zero dynamics.
- Be able to design a stabilizing control law using the input-output linearization method, including
 - 1) Finding the relative degree
 - 2) Writing the system in normal form
 - 3) Creating a linear input-output relation by feedback control
 - 4) Analyzing the zero dynamics
 - 5) Choosing the transformed input variable v to stabilize the origin of the system, locally or globally
- Be able to design a control law that solves the local tracking control problem, using the input-output linearization method
- Be able to discuss the advantages and the disadvantages of the input-output linearization method

Literature

Today's lecture is based on

Khalil **Chapter 13**
 Sections 13.1, 13.2 and 13.4
 Example 13.16 - page 538 is additional material

Motivation



A number of methods exist for control design for linear systems.

It would therefore be nice if we could obtain a linear system instead of the nonlinear system we are dealing with.

Jacobian linearization: An approximation

Question

Is it possible to algebraically transform a nonlinear system dynamics into a (fully or partly) linear one?

Input-state linearization (Full-state linearization)



Given a nonlinear system

$$\dot{x} = f(x) + G(x)u$$

where $f(0) = 0$, and $f: \mathbb{D} \rightarrow \mathbb{R}^n$ and $G: \mathbb{D} \rightarrow \mathbb{R}^{n \times p}$ are sufficiently smooth on a domain $\mathbb{D} \subset \mathbb{R}^n$.

Find a **state transformation**

$$z = T(x)$$

and an **input transformation** (a feedback control)

$$u = \alpha(x) + \beta(x)v$$

such that the corresponding closed-loop system

$$\dot{x} = f(x) + G(x)\alpha(x) + G(x)\beta(x)v$$

written in the coordinates $z = T(x)$ is **linear** and **controllable**.

Input-state linearization (cont.)



i.e.

$$\dot{z} = \frac{dT}{dt}T(x) = \frac{\partial T}{\partial x}\dot{x} = Az + Bv$$

where

$$\left[\frac{\partial T}{\partial x}(f(x) + G(x)\alpha(x)) \right]_{x=T^{-1}(z)} = Az$$

$$\left[\frac{\partial T}{\partial x}G(x)\beta(x) \right]_{x=T^{-1}(z)} = B$$

and

$$\text{rank}[B \quad AB \quad \dots \quad A^{n-1}B] = n$$

Example: Pendulum equation

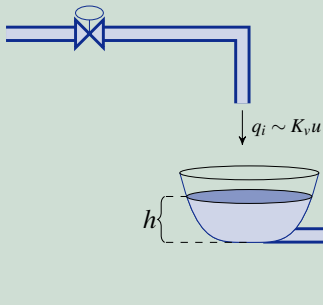
See pages 505 - 507

Example

Fluid level control



Fluid level control



Model:

$$A(x)\dot{x} = -a\sqrt{2gx} + K_v u$$

$$y = x$$

Find:

$$u = \alpha(x) + \beta(x)v$$

$$z = T(x)$$

Find a state transformation $z = T(x)$, an input transformation $u = \alpha(x) + \beta(x)v$, and a control law $v(z)$ which

- asymptotically stabilizes $z_d = h_d = \text{const.}$
- asymptotically tracks $z_d(t) = h_d(t)$



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Input-output linearization

The system



Given a nonlinear system

$$\begin{aligned}\dot{x} &= f(x) + g(x)u \\ y &= h(x)\end{aligned}$$

where $f: \mathbb{D} \rightarrow \mathbb{R}^n$, $g: \mathbb{D} \rightarrow \mathbb{R}^n$ and $h: \mathbb{D} \rightarrow \mathbb{R}$ are sufficiently smooth on a domain $\mathbb{D} \subset \mathbb{R}^n$.

Note: $\dim u = \dim y = 1$



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The method

Notation



Lie derivative of h

$$L_f h = \frac{\partial h}{\partial x} f$$

$$L_f^2 h = L_f (L_f h) = \frac{\partial L_f h}{\partial x} f$$

$$\vdots$$

$$L_f^0 h = h$$

$$L_f^i h = L_f (L_f^{i-1} h), \quad i = 1, 2, \dots$$



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The method

Step 1 - Find the relative degree

Given the system

$$\begin{aligned}\dot{x} &= f(x) + g(x)u \\ y &= h(x)\end{aligned}$$

where $f: \mathbb{D} \rightarrow \mathbb{R}^n$, $g: \mathbb{D} \rightarrow \mathbb{R}^n$ and $h: \mathbb{D} \rightarrow \mathbb{R}$ are sufficiently smooth on a domain $\mathbb{D} \subset \mathbb{R}^n$.

Example:

$$\begin{aligned}\dot{x}_1 &= -x_1 + 2u \\ \dot{x}_2 &= x_3 \\ \dot{x}_3 &= -x_3 + x_2x_3 + u \\ y &= x_2\end{aligned}$$

1) Differentiate y until u appears

$$\begin{aligned}\dot{y} &= \frac{\partial h}{\partial x} \dot{x} = \frac{\partial h}{\partial x} f + \frac{\partial h}{\partial x} g \cdot u \\ &= L_f h + L_g h \cdot u\end{aligned}$$

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Lecture 12: Feedback Linearization

The method

Step 1 - Find the relative degree

Suppose $L_g h = 0 \quad \forall x \in \mathbb{D}_0 \subset \mathbb{D}$

$$\begin{aligned}\ddot{y} &= \frac{\partial(L_f h)}{\partial x} \dot{x} = \underbrace{\frac{\partial(L_f h)}{\partial x} f}_{L_f^2 h} + \underbrace{\frac{\partial(L_f h)}{\partial x} g(x) u}_{L_g L_f h \cdot u} \\ &\vdots \\ y^{(i)} &= L_f^{(i)} h + L_g L_f^{(i-1)} h \cdot u\end{aligned}$$

Relative degree

The system has relative degree ρ in a region $\mathbb{D}_0 \subset \mathbb{D} \subset \mathbb{R}^n$ if

$$\left. \begin{aligned}L_g L_f^{(i-1)} h &= 0, \quad 1 \leq i \leq \rho - 1 \\ L_g L_f^{(\rho-1)} h &\neq 0\end{aligned} \right\} \forall x \in \mathbb{D}_0$$

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The method

Step 1 - Find the relative degree

Terminology/Relation to linear systems

Let

$$\begin{aligned}\dot{x} &= Ax + Bu \\ y &= Cx\end{aligned}$$

with $\dim u = \dim y = 1$.

The system transfer function is

$$h(s) = C(sI - A)^{-1}B = K \frac{s^m + b_{m-1}s^{m-1} + \dots + b_0}{s^n + a_{n-1}s^{n-1} + \dots + a_0}$$

Then $\rho = n - m$

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The method

Step 1 - Find the relative degree



When is input-output linearization possible?

- Q When is it possible to perform an input-output linearization?
- A If the relative degree is well defined in the region of interest \mathbb{D}_0 (Theorem 13.1), then the system can be input-output linearized.

The method

Step 1 - Find the relative degree



Example

Given

$$\begin{aligned}\dot{x}_1 &= x_2 \\ \dot{x}_2 &= -x_1 + x_2 + u\end{aligned}$$

find the relative degree ρ at $x_0 = 0$ when

- a) $y = x_1$
b) $y = x_2^2$

Example

Given

$$\begin{aligned}\dot{x}_1 &= x_2 \\ \dot{x}_2 &= (1 + x_1)u \\ \dot{x}_3 &= u\end{aligned}$$

find the relative degree ρ at $x_0 = 0$ when

- a) $y = x_1$

The method

Step 2 - Write the system in normal form



Suppose ρ is well defined in \mathbb{D}_0

2a) Derive the external dynamics

2a) Let

$$\begin{aligned}\psi_1 &= y \\ \psi_2 &= \dot{y} \\ &\vdots \\ \psi_\rho &= y^{(\rho-1)}\end{aligned}$$

Then

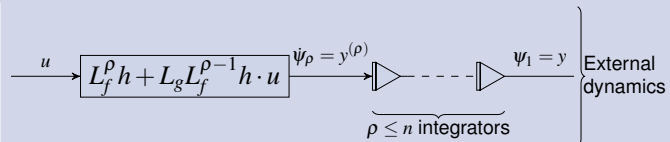
$$\begin{aligned}\dot{\psi}_1 &= \psi_2 \\ \dot{\psi}_2 &= \psi_3 \\ &\vdots \\ \dot{\psi}_\rho &= L_f^\rho h + L_g L_f^{\rho-1} h \cdot u\end{aligned}$$

The method

Step 2a - Derive the external dynamics



The external dynamics



If $\rho < n$

$\phi = ?$ Internal dynamics

The method

Step 2b - Derive the internal dynamics



2b) Derive the internal dynamics

2b) Choose $n - \rho$ coordinates $\phi_1, \dots, \phi_{n-\rho}$

→ Coordinate transformation

$$z = T(x) = \begin{bmatrix} \phi_1 \\ \vdots \\ \phi_{n-\rho} \\ \psi_1 \\ \vdots \\ \psi_\rho \end{bmatrix} \quad \text{normal coordinates/states}$$

The method

Step 2b - Derive the internal dynamics



Choose $\phi_1, \dots, \phi_{n-\rho}$ such that

T is a diffeomorphism

and

$$\begin{aligned} L_g \phi_1 &= 0 \\ &\vdots \\ L_g \phi_{n-\rho} &= 0 \end{aligned}$$

and

$$\phi_i(0) = 0$$

When are these conditions possible to fulfill?

Always possible when the relative degree is well-defined $\rho \leq n$ (Theorem 13.1)

Useful fact

The Jacobian matrix

$$\left. \frac{\partial T}{\partial x} \right|_{x_0}$$

⇒

T is a diffeomorphism in a neighborhood of x_0

is nonsingular

The method

Step 2b - Derive the internal dynamics

$$\begin{aligned}\dot{\phi}_j &= \frac{\partial \phi_j}{\partial x} \dot{x} = L_f \phi_j + \underbrace{L_g \phi_j \cdot u}_{=0} \\ &= f_{0j}(\phi_i, \psi_i)\end{aligned}$$

Write the system in normal form:

$$\left. \begin{aligned}\dot{\phi}_1 &= f_{01}(\phi_i, \psi_i) \\ &\vdots \\ \dot{\phi}_{n-\rho} &= f_{0n-\rho}(\phi_i, \psi_i)\end{aligned} \right\} \begin{array}{l} \text{Internal dynamics} \\ \dot{\phi} = f_0(\phi, \psi) \end{array}$$

$$\left. \begin{aligned}\dot{\psi}_1 &= \psi_2 \\ \dot{\psi}_2 &= \psi_3 \\ &\vdots \\ \dot{\psi}_\rho &= L_f^\rho h + L_g L_f^{\rho-1} h \cdot u\end{aligned} \right\} \text{External dynamics}$$

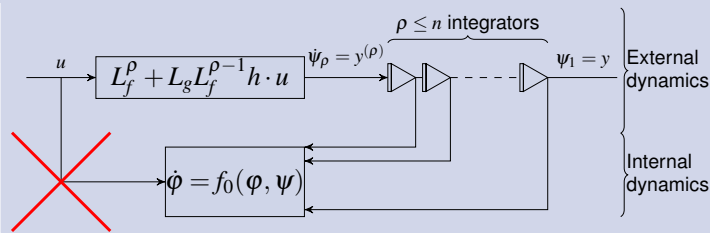
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The method

Step 2 - Write the system in normal form

The system in normal form



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Lecture 12: Feedback Linearization

The method

Step 3 - Choose u to cancel the nonlinearities3) Choose u to cancel the nonlinearities

Create a linear input-output relation by feedback control:

Choose u to cancel the nonlinearities

(an input transformation/ a linearizing inner feedback control loop)

$$u = \frac{1}{L_g L_f^{\rho-1} h} (-L_f^\rho h + v)$$

$$\Downarrow$$

$$\begin{aligned}\dot{\phi} &= f_0(\phi, \psi) \\ \left. \begin{aligned}\dot{\psi}_1 &= \psi_2 \\ &\vdots \\ \dot{\psi}_\rho &= v\end{aligned} \right\} y^{(\rho)} = v\end{aligned}$$

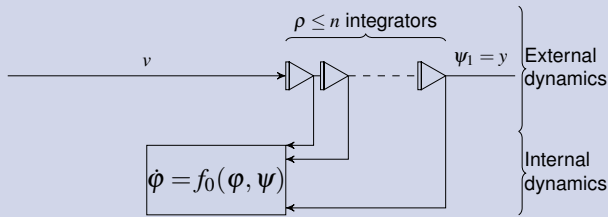
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The method

Step 3 - Choose u to cancel the nonlinearities

A linear input-output relationship



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Lecture 12: Feedback Linearization

The method

Step 4 - Analyze the zero-dynamics



Definition: Zero-dynamics

The zero-dynamics is the internal dynamics when the system output is kept at zero by the input

i.e.

$$y \equiv 0$$

$$\vdots$$

$$y^{(\rho)} \equiv 0 \quad (\text{i.e. } u_0(x) = -\frac{L_f^\rho h(x)}{L_g L_f^{\rho-1} h(x)})$$

$$\Updownarrow$$

$$\psi_i = 0 \quad i = 1, \dots, \rho$$

The zero-dynamics:

$$\dot{\phi} = f_0(\phi, 0)$$



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The method

Step 4 - Analyze the zero-dynamics



Definition: Minimum phase systems

The system

$$\dot{x} = f(x) + g(x)u$$

$$y = h(x)$$

is minimum phase if (the origin of) the zero-dynamics is asymptotically stable.

Terminology - Relation to linear systems

$$\begin{aligned} \dot{x} &= Ax + Bu \\ y &= Cx \end{aligned} \rightarrow h(s) = K \frac{(s - z_1) \cdots (s - z_m)}{(s - p_1) \cdots (s - p_n)}$$

Zero-dynamics:

$$\dot{\phi} = Q\phi, \quad \lambda_j(Q) = z_j, \quad j = 1, \dots, m = n - \rho$$



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The method

Step 5 - Choose v to solve the control problem

Asymptotic stabilization

Control objective:

 $x = 0$ asymptotically stable $z = \begin{bmatrix} \varphi \\ \psi \end{bmatrix} = 0$ asymptotically stable

We have

$$\dot{\varphi} = f_0(\varphi, \psi)$$

$$\dot{\psi}_1 = \psi_2$$

 \vdots

$$\dot{\psi}_\rho = v$$

Special case of

$$\dot{\varphi} = f_0(\varphi, \psi)$$

$$\dot{\psi} = A\psi + Bv$$

 (A, B) controllable

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Lecture 12: Feedback Linearization

The method

Step 5 - Choose v to solve the control problem

Recall: We have a special case of

$$\dot{\varphi} = f_0(\varphi, \psi)$$

$$\dot{\psi} = A\psi + Bv$$

 (A, B) controllable

$$v = -K\psi \quad K \text{ is chosen such that } (A - BK) \text{ is Hurwitz}$$



$$\dot{\varphi} = f_0(\varphi, \psi)$$

$$\dot{\psi} = (A - BK)\psi \quad \text{Exponentially stable}$$



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Lecture 12: Feedback Linearization

The method

Step 5 - Choose v to solve the control problem

Recall:

$$\dot{\varphi} = f_0(\varphi, \psi)$$

$$\dot{\psi} = (A - BK)\psi \quad \text{Exponentially stable}$$

Lemma 13.1

If the origin $\varphi = 0$ of $\dot{\varphi} = f_0(\varphi, 0)$ is asymptotically stable, then the origin $(\varphi, \psi) = (0, 0)$ of

$$\dot{\varphi} = f_0(\varphi, \psi)$$

$$\dot{\psi} = (A - BK)\psi$$

is asymptotically stable.

NB

Only *local* asymptotic stability can be concluded

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Lecture 12: Feedback Linearization

The method

Step 5 - Choose v to solve the control problem

Summary Step 5 - Asymptotic stabilization

Choose

$$v = -K\psi$$

such that $(A - BK)$ is Hurwitz.

Choose for instance

$$\begin{aligned} v &= -k_0\psi_1 - k_1\psi_2 - \dots - k_{p-1}\psi_p \\ &= -k_0y - k_1\dot{y} - \dots - k_{p-1}y^{(p-1)} \end{aligned}$$

such that

$$s^p + k_{p-1}s^{p-1} + \dots + k_1s + k_0$$

has all its roots strictly in the left-half plane.

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The method

Summary: Input-output linearization for stabilization

If the system

$$\begin{aligned} \dot{x} &= f(x) + g(x)u \\ y &= h(x) \end{aligned}$$

where $f: \mathbb{D} \rightarrow \mathbb{R}^n$, $g: \mathbb{D} \rightarrow \mathbb{R}^n$ and $h: \mathbb{D} \rightarrow \mathbb{R}$ are sufficiently smooth on a domain $\mathbb{D} \subset \mathbb{R}^n$, has a **well-defined relative degree** $\rho \in \mathbb{D}_0 \subset \mathbb{D}$, $1 \leq \rho \leq n$ and is **minimum phase** then the control law

$$u = \frac{1}{L_g L_f^{\rho-1} h} (-L_f^\rho h + v)$$

makes $x = 0$ locally asymptotically stable.

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The method

Input-output linearization for stabilization - global result

Global result (Lemma 13.2)

If $\varphi = f_0(\varphi, \psi)$ is **ISS** then the origin $(\varphi, \psi) = (0, 0)$ of

$$\begin{aligned} \dot{\varphi} &= f_0(\varphi, \psi) \\ \dot{\psi} &= (A - BK)\psi \end{aligned}$$

is globally asymptotically stable GAS.

Proof: Satisfies conditions of Lemma 4.7 (Cascaded control)

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The method

Step 5 - Choose v to solve the control problem

Tracking control

Reference trajectory: $y_d(t), \dot{y}_d(t), \dots, y_d^{(\rho)}$ bounded
 $y_d^{(\rho)}$ piecewise continuous

Define

$$e = \begin{bmatrix} \psi_1 - y_d \\ \vdots \\ \psi_\rho - y_d^{(\rho-1)} \end{bmatrix} = \begin{bmatrix} y - y_d \\ \dot{y} - \dot{y}_d \\ \vdots \\ y^{(\rho-1)} - y_d^{(\rho-1)} \end{bmatrix} = \psi - R, \quad R = \begin{bmatrix} y_d \\ \dot{y}_d \\ \vdots \\ y_d^{(\rho-1)} \end{bmatrix}$$

Choose

$$v = -Ke + y_d^{(\rho)}$$

where K is chosen such that $(A - BK)$ is Hurwitz.

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The method

Step 5 - Choose v

We obtain

$$\begin{aligned} \dot{\phi} &= f_0(\phi, e + R) \\ \dot{e} &= (A - BK)e \quad \text{exponentially stable} \end{aligned}$$

If the origin $\phi = 0$ of $\dot{\phi} = f_0(\phi, 0)$ is asymptotically stable, then, for sufficiently small $e(0)$, $\phi(0)$ and $R(t)$

$$\begin{aligned} e(t) &\xrightarrow{\text{exp}} 0 \\ \phi(t) &\text{ is bounded} \end{aligned}$$

i.e. solves local tracking control problem.

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The method

Step 5 - Choose v to solve the control problem

Summary Step 5 - Tracking control

Choose

$$v = -Ke + y_d^{(\rho)}$$

such that $(A - BK)$ is Hurwitz.

Choose for instance

$$\begin{aligned} v &= -k_0 e_1 - k_1 e_2 - \dots - k_{\rho-1} e_\rho + y_d^{(\rho)} \\ &= -k_0 (y - y_d) - k_1 (\dot{y} - \dot{y}_d) - \dots - k_{\rho-1} (y^{(\rho-1)} - y_d^{(\rho-1)}) + y_d^{(\rho)} \end{aligned}$$

such that

$$s^\rho + k_{\rho-1} s^{\rho-1} + \dots + k_1 s + k_0$$

has all its roots strictly in the left-half plane.

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Lecture 12: Feedback Linearization

The method

Summary: Input-output linearization for tracking control

If the system

$$\begin{aligned}\dot{x} &= f(x) + g(x)u \\ y &= h(x)\end{aligned}$$

where $f: \mathbb{D} \rightarrow \mathbb{R}^n$, $g: \mathbb{D} \rightarrow \mathbb{R}^n$ and $h: \mathbb{D} \rightarrow \mathbb{R}$ are sufficiently smooth on a domain $\mathbb{D} \subset \mathbb{R}^n$, has a **well-defined relative degree** $\rho \in \mathbb{D}_0 \subset \mathbb{D}$, $1 \leq \rho \leq n$ and is **minimum phase** then the control law

$$u = \frac{1}{L_g L_f^{\rho-1} h} (-L_f^\rho h + v)$$

ensures that if $e(0)$ and $\varphi(0)$ and $R(t)$ are sufficiently small, then

$$\begin{aligned}e(t) &\xrightarrow{\exp} 0 \\ \varphi(t) &\text{ is bounded}\end{aligned}$$

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The method

Input-output linearization for tracking control - Global results

If $\rho = n$ then no internal dynamics

System dynamics is then

$$\dot{e} = (A - BK)e \quad \text{GES}$$

If $\dot{\varphi} = f_0(\varphi, \psi)$ is **ISS** then

$$u = \frac{1}{L_g L_f^{\rho-1} h} (-L_f^\rho h + v)$$

gives

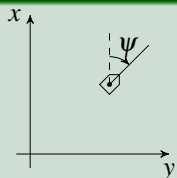
$$\begin{aligned}e(t) &\xrightarrow{\exp} 0 \\ \varphi(t) &\text{ is bounded}\end{aligned} \quad \forall e(0), \varphi(0), R(t)$$

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Example: Dynamic positioning system for ships

Dynamic positioning system for ships



$$\eta = \begin{bmatrix} x \\ y \\ \psi \end{bmatrix}$$

System model:

$$\begin{aligned}M(\eta)\ddot{\eta} + C(\eta, \dot{\eta})\dot{\eta} + D(\eta)\dot{\eta} &= \tau \\ y &= \eta\end{aligned}$$

System properties:

$$\begin{aligned}M &= M^T > 0 \\ z^T D z &> 0 \quad z \neq 0 \\ z^T (\tfrac{1}{2}\dot{M} - C)z &= 0 \quad \forall z \in \mathbb{R}^3\end{aligned}$$

Control problem: Design a control law $\tau = g(t, (\eta, \dot{\eta}))$ using input-output linearization, that makes the origin $(\eta, \dot{\eta}) = (0, 0)$ an asymptotically stable equilibrium point.

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Lecture 12: Feedback Linearization

Advantages/shortcomings



Advantages/shortcomings

- Cancels all dynamics $L_f h$
 - ÷ Does not take advantage of stabilizing terms
 - ÷ Robustness to modelling errors is questionable
- Requires well-defined relative degree
- Requires minimum phase system
- + Exponential convergence
- + We can use linear control design methods
- + Easy tuning

Next lecture



Backstepping

Recommended reading:

Khalil **Chapter 14**
Sections 14.3 pages 589-598