

# TTK4150 Nonlinear Control Systems

## Lecture 12

Feedback Linearization



# Previous lectures on nonlinear control design



## Previous lectures on nonlinear control design:

- Lyapunov based control design
- Cascaded control: Lemma 4.7 allows for modular design  
(And background material, Sontag and Loria)
- Passivity-based control design

# Outline I



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# Outline II



- Summary: Input-output linearization for stabilization
- Summary: Input-output linearization for tracking control
- Application example
- Advantages/shortcomings

# Today's goals



## After today you should...

- Know the concepts of relative degree, normal form, external dynamics, internal dynamics and zero dynamics.
- Be able to design a stabilizing control law using the input-output linearization method, including
  - 1) Finding the relative degree
  - 2) Writing the system in normal form
  - 3) Creating a linear input-output relation by feedback control
  - 4) Analyzing the zero dynamics
  - 5) Choosing the transformed input variable  $v$  to stabilize the origin of the system, locally or globally
- Be able to design a control law that solves the local tracking control problem, using the input-output linearization method
- Be able to discuss the advantages and the disadvantages of the input-output linearization method





Today's lecture is based on

Khalil **Chapter 13**

Sections 13.1, 13.2 and 13.4

Example 13.16 - page 538 is additional material

# Motivation



A number of methods exist for control design for linear systems.

It would therefore be nice if we could obtain a linear system instead of the nonlinear system we are dealing with.

Jacobian linearization: An approximation

## Question

Is it possible to algebraically transform a nonlinear system dynamics into a (fully or partly) linear one?

# Input-state linearization (Full-state linearization)



Given a nonlinear system

$$\dot{x} = f(x) + G(x)u$$

where  $f(0) = 0$ , and  $f : \mathbb{D} \rightarrow \mathbb{R}^n$  and  $G : \mathbb{D} \rightarrow \mathbb{R}^{n \times p}$  are sufficiently smooth on a domain  $\mathbb{D} \subset \mathbb{R}^n$ .

Find a **state transformation**

$$z = T(x)$$

and an **input transformation** (a feedback control)

$$u = \alpha(x) + \beta(x)v$$

such that the corresponding closed-loop system

$$\dot{x} = f(x) + G(x)\alpha(x) + G(x)\beta(x)v$$

written in the coordinates  $z = T(x)$  is **linear** and **controllable**





# Input-state linearization (cont.)



i.e.

$$\dot{z} = \frac{d}{dt}T(x) = \frac{\partial T}{\partial x}\dot{x} = Az + Bv$$

where

$$\begin{aligned}\left[\frac{\partial T}{\partial x}(f(x) + G(x)\alpha(x))\right]_{x=T^{-1}(z)} &= Az \\ \left[\frac{\partial T}{\partial x}G(x)\beta(x)\right]_{x=T^{-1}(z)} &= B\end{aligned}$$

and

$$\text{rank}[B \quad AB \quad \dots \quad A^{n-1}B] = n$$

Example: Pendulum equation

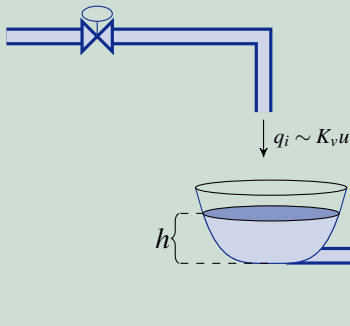
See pages 505 - 507

# Example

## Fluid level control



### Fluid level control



Model:

$$A(x)\dot{x} = -a\sqrt{2gx} + K_v u$$

$$y = x$$

Find:

$$u = \alpha(x) + \beta(x)v$$

$$z = T(x)$$

Find a state transformation  $z = T(x)$ , an input transformation  $u = \alpha(x) + \beta(x)v$ , and a control law  $v(z)$  which

- asymptotically stabilizes  $z_d = h_d = \text{const.}$
- asymptotically tracks  $z_d(t) = h_d(t)$

# Input-output linearization

## The system



Given a nonlinear system

$$\begin{aligned}\dot{x} &= f(x) + g(x)u \\ y &= h(x)\end{aligned}$$

where  $f: \mathbb{D} \rightarrow \mathbb{R}^n$ ,  $g: \mathbb{D} \rightarrow \mathbb{R}^n$  and  $h: \mathbb{D} \rightarrow \mathbb{R}$  are sufficiently smooth on a domain  $\mathbb{D} \subset \mathbb{R}^n$ .

Note:  $\dim u = \dim y = 1$

# The method

## Notation



### Lie derivative of $h$

$$L_f h = \frac{\partial h}{\partial x} f$$

$$L_f^2 h = L_f (L_f h) = \frac{\partial L_f h}{\partial x} f$$

$$\vdots$$

$$L_f^0 h = h$$

$$L_f^i h = L_f (L_f^{i-1} h), \quad i = 1, 2, \dots$$

# The method

## Step 1 - Find the relative degree



Given the system

$$\begin{aligned}\dot{x} &= f(x) + g(x)u \\ y &= h(x)\end{aligned}$$

where  $f: \mathbb{D} \rightarrow \mathbb{R}^n$ ,  $g: \mathbb{D} \rightarrow \mathbb{R}^n$  and  $h: \mathbb{D} \rightarrow \mathbb{R}$  are sufficiently smooth on a domain  $\mathbb{D} \subset \mathbb{R}^n$ .

### Example:

$$\dot{x}_1 = -x_1 + 2u$$

$$\dot{x}_2 = x_3$$

$$\dot{x}_3 = -x_3 + x_2 x_3 + u$$

$$y = x_2$$

### 1) Differentiate $y$ until $u$ appears

$$\begin{aligned}\dot{y} &= \frac{\partial h}{\partial x} \dot{x} = \frac{\partial h}{\partial x} f + \frac{\partial h}{\partial x} g \cdot u \\ &= L_f h + L_g h \cdot u\end{aligned}$$

# The method

## Step 1 - Find the relative degree

Suppose  $L_g h = 0 \quad \forall x \in \mathbb{D}_0 \subset \mathbb{D}$

$$\ddot{y} = \frac{\partial(L_f h)}{\partial x} \dot{x} = \underbrace{\frac{\partial(L_f h)}{\partial x} f}_{L_f^2 h} + \underbrace{\frac{\partial(L_f h)}{\partial x} g(x) u}_{L_g L_f h \cdot u}$$

$\vdots$

$$y^{(i)} = L_f^{(i)} h + L_g L_f^{(i-1)} h \cdot u$$

## Relative degree

The system has relative degree  $\rho$  in a region  $\mathbb{D}_0 \subset \mathbb{D} \subset \mathbb{R}^n$  if

$$\left. \begin{array}{l} L_g L_f^{(i-1)} h = 0, \quad 1 \leq i \leq \rho - 1 \\ L_g L_f^{(\rho-1)} h \neq 0 \end{array} \right\} \forall x \in \mathbb{D}_0$$

# The method

## Step 1 - Find the relative degree



### Terminology/Relation to linear systems

Let

$$\dot{x} = Ax + Bu$$

$$y = Cx$$

with  $\dim u = \dim y = 1$ .

The system transfer function is

$$h(s) = C(sI - A)^{-1}B = K \frac{s^m + b_{m-1}s^{m-1} + \dots + b_0}{s^n + a_{n-1}s^{n-1} + \dots + a_0}$$

Then  $\rho = n - m$

# The method

## Step 1 - Find the relative degree



### When is input-output linearization possible?

**Q** When is it possible to perform an input-output linearization?

**A** If the relative degree is well defined in the region of interest  $\mathbb{D}_0$  (Theorem 13.1), then the system can be input-output linearized.



# The method

## Step 1 - Find the relative degree



### Example

Given

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = -x_1 + x_2 + u$$

find the relative degree  $\rho$  at  $x_0 = 0$  when

a)  $y = x_1$

b)  $y = x_2^2$

### Example

Given

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = (1 + x_1)u$$

$$\dot{x}_3 = u$$

find the relative degree  $\rho$  at  $x_0 = 0$  when

a)  $y = x_1$

# The method

## Step 2 - Write the system in normal form



Suppose  $\rho$  is well defined in  $\mathbb{D}_0$

### 2a) Derive the external dynamics

2a) Let

$$\psi_1 = y$$

$$\psi_2 = \dot{y}$$

$$\vdots$$

$$\psi_\rho = y^{(\rho-1)}$$

Then

$$\dot{\psi}_1 = \psi_2$$

$$\dot{\psi}_2 = \psi_3$$

$$\vdots$$

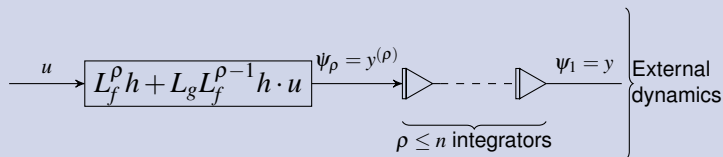
$$\dot{\psi}_\rho = L_f^\rho h + L_g L_f^{\rho-1} h \cdot u$$

# The method

## Step 2a - Derive the external dynamics



### The external dynamics



If  $\rho < n$

$\dot{\phi} = ?$

Internal dynamics

# The method

## Step 2b - Derive the internal dynamics



### 2b) Derive the internal dynamics

2b) Choose  $n - \rho$  coordinates  $\varphi_1, \dots, \varphi_{n-\rho}$

→ Coordinate transformation

$$z = T(x) = \begin{bmatrix} \varphi_1 \\ \vdots \\ \varphi_{n-\rho} \\ \psi_1 \\ \vdots \\ \psi_\rho \end{bmatrix} \quad \text{normal coordinates/states}$$

# The method

## Step 2b - Derive the internal dynamics



Choose  $\varphi_1, \dots, \varphi_{n-\rho}$  such that

T is a  
diffeomorphism

and

$$\begin{aligned} L_g \varphi_1 &= 0 \\ &\vdots \\ L_g \varphi_{n-\rho} &= 0 \end{aligned}$$

and

$$\varphi_i(0) = 0$$

When are these conditions possible to fulfill?

Always possible when the relative degree is well-defined  $\rho \leq n$   
(Theorem 13.1)

### Useful fact

The Jacobian matrix

$$\left. \frac{\partial T}{\partial x} \right|_{x_0}$$

$\Rightarrow$

T is a  
diffeomorphism in  
a neighborhood  
of  $x_0$

is nonsingular

# The method

## Step 2b - Derive the internal dynamics



$$\begin{aligned}\dot{\phi}_j &= \frac{\partial \phi_j}{\partial x} \dot{x} = L_f \phi_j + \underbrace{L_g \phi_j \cdot u}_{=0} \\ &= f_{0j}(\phi_i, \psi_i)\end{aligned}$$

Write the system in normal form:

$$\left. \begin{aligned}\dot{\phi}_1 &= f_{01}(\phi_i, \psi_i) \\ &\vdots \\ \dot{\phi}_{n-\rho} &= f_{0_{n-\rho}}(\phi_i, \psi_i)\end{aligned} \right\} \begin{array}{l} \text{Internal dynamics} \\ \dot{\phi} = f_0(\phi, \psi) \end{array}$$
  

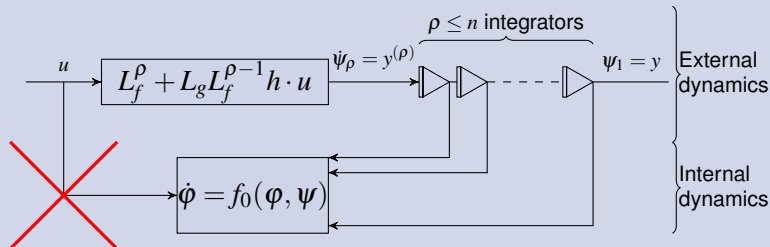
$$\left. \begin{aligned}\dot{\psi}_1 &= \psi_2 \\ \dot{\psi}_2 &= \psi_3 \\ &\vdots \\ \dot{\psi}_\rho &= L_f^\rho h + L_g L_f^{\rho-1} h \cdot u\end{aligned} \right\} \text{External dynamics}$$

# The method

## Step 2 - Write the system in normal form



### The system in normal form



# The method

## Step 3 - Choose $u$ to cancel the nonlinearities



### 3) Choose $u$ to cancel the nonlinearities

Create a linear input-output relation by feedback control:

Choose  $u$  to cancel the nonlinearities

(an input transformation/ a linearizing inner feedback control loop)

$$u = \frac{1}{L_g L_f^{\rho-1} h} (-L_f^{\rho} h + v)$$



$$\dot{\phi} = f_0(\phi, \psi)$$

$$\left. \begin{array}{l} \dot{\psi}_1 = \psi_2 \\ \vdots \\ \dot{\psi}_{\rho} = v \end{array} \right\} y^{(\rho)} = v$$

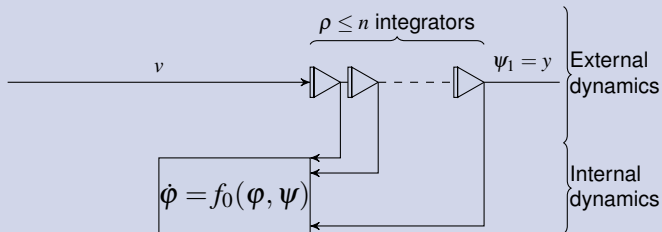


# The method

## Step 3 - Choose $u$ to cancel the nonlinearities



### A linear input-output relationship



# The method

## Step 4 - Analyze the zero-dynamics



### Definition: Zero-dynamics

The zero-dynamics is the internal dynamics when the system output is kept at zero by the input

i.e.

$$y \equiv 0$$

$$\vdots$$

$$y^{(\rho)} \equiv 0 \quad (\text{i.e.} \quad u_0(x) = -\frac{L_f^\rho h(x)}{L_g L_f^{\rho-1} h(x)})$$

$$\Updownarrow$$

$$\psi_i = 0 \quad i = 1, \dots, \rho$$

### The zero-dynamics:

$$\dot{\phi} = f_0(\phi, 0)$$



# The method

## Step 4 - Analyze the zero-dynamics



### Definition: Minimum phase systems

The system

$$\begin{aligned}\dot{x} &= f(x) + g(x)u \\ y &= h(x)\end{aligned}$$

is minimum phase if (the origin of) the zero-dynamics is asymptotically stable.

### Terminology - Relation to linear systems

$$\begin{aligned}\dot{x} &= Ax + Bu \\ y &= Cx\end{aligned} \rightarrow h(s) = K \frac{(s - z_1) \cdots (s - z_m)}{(s - p_1) \cdots (s - p_n)}$$

Zero-dynamics:

$$\dot{\phi} = Q\phi, \quad \lambda_j(Q) = z_j, \quad j = 1, \dots, m = n - \rho$$



# The method

Step 5 - Choose  $v$  to solve the control problem



## Asymptotic stabilization

Control objective:

$x = 0$  asymptotically stable



$z = \begin{bmatrix} \varphi \\ \psi \end{bmatrix} = 0$  asymptotically stable

We have

$$\begin{aligned} \dot{\varphi} &= f_0(\varphi, \psi) \\ \dot{\psi}_1 &= \psi_2 \\ &\vdots \\ \dot{\psi}_\rho &= v \end{aligned}$$

Special case of

$$\begin{aligned} \dot{\varphi} &= f_0(\varphi, \psi) \\ \dot{\psi} &= A\psi + Bv \end{aligned}$$

$(A, B)$  controllable

# The method

## Step 5 - Choose $v$ to solve the control problem



**Recall:** We have a special case of

$$\begin{aligned}\dot{\phi} &= f_0(\phi, \psi) \\ \dot{\psi} &= A\psi + Bv\end{aligned}$$

$(A, B)$  controllable

$$v = -K\psi \quad K \text{ is chosen such that } (A - BK) \text{ is Hurwitz}$$



$$\dot{\phi} = f_0(\phi, \psi)$$

$$\dot{\psi} = (A - BK)\psi \quad \} \text{ Exponentially stable}$$

# The method

## Step 5 - Choose $v$ to solve the control problem

**Recall:**

$$\dot{\phi} = f_0(\phi, \psi)$$

$$\dot{\psi} = (A - BK)\psi \} \text{ Exponentially stable}$$

### Lemma 13.1

If the origin  $\phi = 0$  of  $\dot{\phi} = f_0(\phi, 0)$  is asymptotically stable, then the origin  $(\phi, \psi) = (0, 0)$  of

$$\dot{\phi} = f_0(\phi, \psi)$$

$$\dot{\psi} = (A - BK)\psi$$

is asymptotically stable.

**NB**

Only *local* asymptotic stability can be concluded

# The method

## Step 5 - Choose $v$ to solve the control problem



### Summary Step 5 - Asymptotic stabilization

Choose

$$v = -K\psi$$

such that  $(A - BK)$  is Hurwitz.

Choose for instance

$$\begin{aligned} v &= -k_0\psi_1 - k_1\psi_2 - \cdots - k_{\rho-1}\psi_{\rho} \\ &= -k_0y - k_1\dot{y} - \cdots - k_{\rho-1}y^{(\rho-1)} \end{aligned}$$

such that

$$s^{\rho} + k_{\rho-1}s^{\rho-1} + \cdots + k_1s + k_0$$

has all its roots strictly in the left-half plane.

# The method

## Summary: Input-output linearization for stabilization



If the system

$$\begin{aligned}\dot{x} &= f(x) + g(x)u \\ y &= h(x)\end{aligned}$$

where  $f : \mathbb{D} \rightarrow \mathbb{R}^n$ ,  $g : \mathbb{D} \rightarrow \mathbb{R}^n$  and  $h : \mathbb{D} \rightarrow \mathbb{R}$  are sufficiently smooth on a domain  $\mathbb{D} \subset \mathbb{R}^n$ , has a **well-defined relative degree**  $\rho \in \mathbb{D}_0 \subset \mathbb{D}$ ,  $1 \leq \rho \leq n$  and is **minimum phase** then the control law

$$u = \frac{1}{L_g L_f^{\rho-1} h} (-L_f^\rho h + v)$$

makes  $x = 0$  locally asymptotically stable.



# The method

## Input-output linearization for stabilization - global result



### Global result (Lemma 13.2)

If  $\varphi = f_0(\varphi, \psi)$  is **ISS** then the origin  $(\varphi, \psi) = (0, 0)$  of

$$\begin{aligned}\dot{\varphi} &= f_0(\varphi, \psi) \\ \dot{\psi} &= (A - BK)\psi\end{aligned}$$

is globally asymptotically stable GAS.

Proof: Satisfies conditions of Lemma 4.7 (Cascaded control)

# The method

## Step 5 - Choose $v$ to solve the control problem

### Tracking control

Reference trajectory:  $y_d(t), \dot{y}_d(t), \dots, y_d^{(\rho)}(t)$  bounded  
 $y_d^{(\rho)}$  piecewise continuous

Define

$$e = \begin{bmatrix} \psi_1 - y_d \\ \vdots \\ \psi_\rho - y_d^{(\rho-1)} \end{bmatrix} = \begin{bmatrix} y - y_d \\ \dot{y} - \dot{y}_d \\ \vdots \\ y^{(\rho-1)} - y_d^{(\rho-1)} \end{bmatrix} = \psi - R, \quad R = \begin{bmatrix} y_d \\ \dot{y}_d \\ \vdots \\ y_d^{(\rho-1)} \end{bmatrix}$$

Choose

$$v = -Ke + y_d^{(\rho)}$$

where  $K$  is chosen such that  $(A - BK)$  is Hurwitz.

# The method

## Step 5 - Choose v



We obtain

$$\begin{aligned}\dot{\varphi} &= f_0(\varphi, e + R) \\ \dot{e} &= (A - BK)e \quad \text{exponentially stable}\end{aligned}$$

If the origin  $\varphi = 0$  of  $\dot{\varphi} = f_0(\varphi, 0)$  is asymptotically stable, then, for sufficiently small  $e(0)$ ,  $\varphi(0)$  and  $R(t)$

$$\begin{aligned}e(t) &\xrightarrow{\text{exp}} 0 \\ \varphi(t) &\text{ is bounded}\end{aligned}$$

i.e. solves local tracking control problem.

# The method

## Step 5 - Choose $v$ to solve the control problem



### Summary Step 5 - Tracking control

Choose

$$v = -Ke + y_d^{(\rho)}$$

such that  $(A - BK)$  is Hurwitz.

Choose for instance

$$\begin{aligned} v &= -k_0 e_1 - k_1 e_2 - \dots - k_{\rho-1} e_{\rho} + y_d^{(\rho)} \\ &= -k_0 (y - y_d) - k_1 (\dot{y} - \dot{y}_d) - \dots - k_{\rho-1} (y^{(\rho-1)} - y_d^{(\rho-1)}) + y_d^{(\rho)} \end{aligned}$$

such that

$$s^{\rho} + k_{\rho-1} s^{\rho-1} + \dots + k_1 s + k_0$$

has all its roots strictly in the left-half plane.



# The method

Summary: Input-output linearization for tracking control



If the system

$$\begin{aligned}\dot{x} &= f(x) + g(x)u \\ y &= h(x)\end{aligned}$$

where  $f: \mathbb{D} \rightarrow \mathbb{R}^n$ ,  $g: \mathbb{D} \rightarrow \mathbb{R}^n$  and  $h: \mathbb{D} \rightarrow \mathbb{R}$  are sufficiently smooth on a domain  $\mathbb{D} \subset \mathbb{R}^n$ , has a **well-defined relative degree**  $\rho \in \mathbb{D}_0 \subset \mathbb{D}$ ,  $1 \leq \rho \leq n$  and is **minimum phase** then the control law

$$u = \frac{1}{L_g L_f^{\rho-1} h} (-L_f^\rho h + v)$$

ensures that if  $e(0)$  and  $\varphi(0)$  and  $R(t)$  are sufficiently small, then

$$\begin{aligned}e(t) &\xrightarrow{\exp} 0 \\ \varphi(t) &\text{ is bounded}\end{aligned}$$

# The method

## Input-output linearization for tracking control - Global results



If  $\rho = n$  then no internal dynamics



System dynamics is then

$$\dot{e} = (A - BK)e \quad \text{GES}$$

If  $\dot{\phi} = f_0(\phi, \psi)$  is **ISS** then

$$u = \frac{1}{L_g L_f^{\rho-1} h} (-L_f^{\rho} h + v)$$

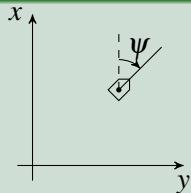
gives

$$\begin{aligned} e(t) &\xrightarrow{\exp} 0 \\ \phi(t) &\text{ is bounded} \end{aligned} \quad \forall e(0), \phi(0), R(t)$$

# Example: Dynamic positioning system for ships



## Dynamic positioning system for ships



$$\eta = \begin{bmatrix} x \\ y \\ \psi \end{bmatrix}$$

### System model:

$$M(\eta)\ddot{\eta} + C(\eta, \dot{\eta})\dot{\eta} + D(\eta)\dot{\eta} = \tau$$

$$y = \eta$$

### System properties:

$$M = M^T > 0$$

$$z^T D z > 0 \quad z \neq 0$$

$$z^T \left( \frac{1}{2} \dot{M} - C \right) z = 0 \quad \forall z \in \mathbb{R}^3$$

**Control problem:** Design a control law  $\tau = g(t, (\eta, \dot{\eta}))$  using input-output linearization, that makes the origin  $(\eta, \dot{\eta}) = (0, 0)$  an asymptotically stable equilibrium point.

# Advantages/shortcomings



## Advantages/shortcomings

- Cancels all dynamics  $L_f h$ 
  - ÷ Does not take advantage of stabilizing terms
  - ÷ Robustness to modelling errors is questionable
- Requires well-defined relative degree
- Requires minimum phase system
- + Exponential convergence
- + We can use linear control design methods
- + Easy tuning



# Next lecture



## Backstepping

Recommended reading:

Khalil **Chapter 14**  
Sections 14.3 pages 589-598