

# 034IN - FONDAMENTI DI AUTOMATICA - FUNDAMENTALS OF AUTOMATIC CONTROL

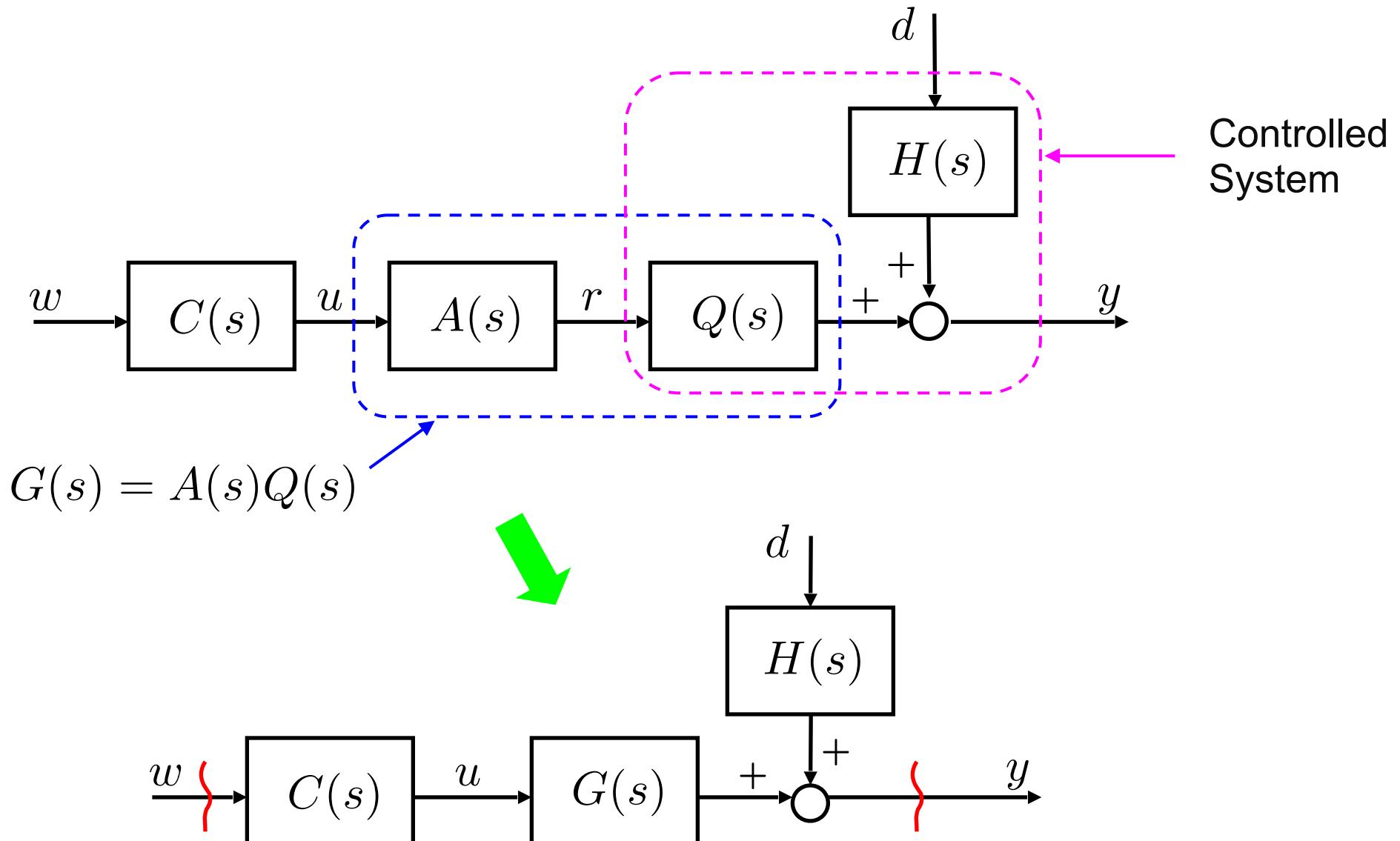
## A.Y. 2023-2024

### Part IX: Analysis of Feedback Control Systems

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# Open-Loop Control Systems





$$y(t) = w(t), t \geq 0 \quad \rightarrow \quad \frac{Y(s)}{W(s)} = 1 \quad \rightarrow \quad C(s)G(s) = 1$$



$$C(s) = G(s)^{-1}$$

The “ideal” open-loop controller “inverts” the system’s dynamics

Hence:

- pole-zero “cancellations” in the right half-plane may occur causing **unstable hidden dynamics**
- open-loop stabilisation of unstable systems is not possible
- the controller  $C(s)$  may result in having **more zeros than poles**, thus not being physically implementable
- **uncertainty** in  $G(s)$  makes the ideal performance anyway not achievable

## Examples



Consider the controlled system  $G(s) = \frac{10(1 + s)}{(1 + 2s)(1 + 0.1s)}$

↳  $C_0(s) = \frac{0.1(1 + 2s)(1 + 0.1s)}{1 + s}$

The “ideal” open-loop controller shows two zeros and one pole and hence it is not physically implementable

- Consider a first candidate of physically implementable open-loop controller:

$$C_1(s) = \frac{0.1(1 + 2s)(1 + 0.1s)}{(1 + s)(1 + 0.01s)}$$

↳  $F_1(s) = \frac{Y(s)}{W(s)} = C_1(s)G(s) = \frac{1}{1 + 0.01s}$

Low-pass filter  
with bandwidth  
 $B_1 \simeq [0, 100]$

- Consider a second **simpler** candidate of physically implementable open-loop controller:

$$C_2(s) = \frac{0.1(1 + 2s)}{1 + s}$$

↳  $F_2(s) = \frac{Y(s)}{W(s)} = C_2(s)G(s) = \frac{1}{1 + 0.1s}$

Low-pass filter  
with bandwidth  
 $B_2 \simeq [0, 10]$

- Consider a third **much simpler (not even dynamic)** candidate of physically implementable open-loop controller:

$$C_3(s) = 0.1$$

↳  $F_3(s) = \frac{Y(s)}{W(s)} = C_3(s)G(s) = \frac{1 + s}{(1 + 2s)(1 + 0.1s)}$

Low-pass filter  
with bandwidth  
 $B_3 \simeq [0, 0.5]$

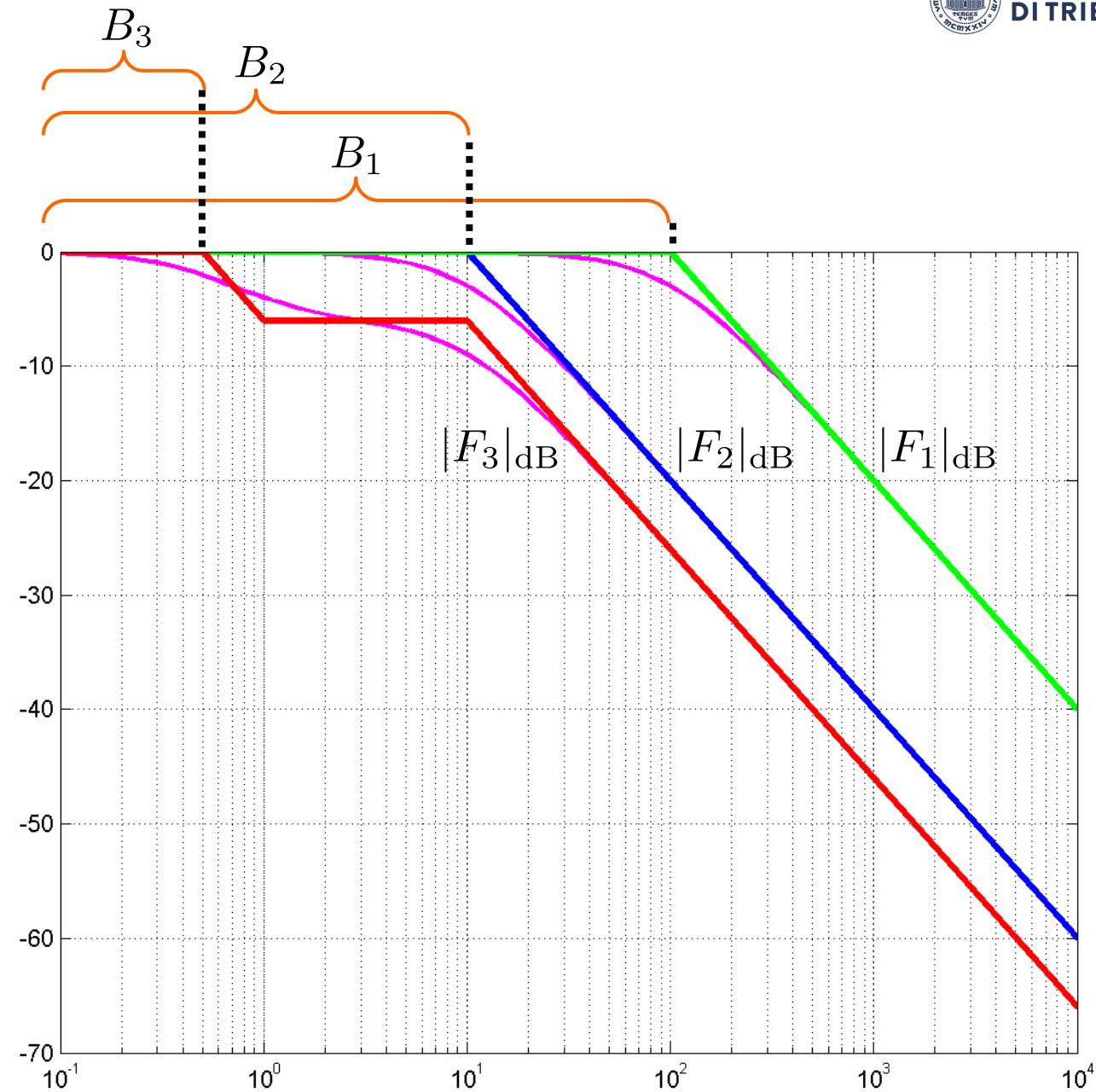
## Examples - Comparison



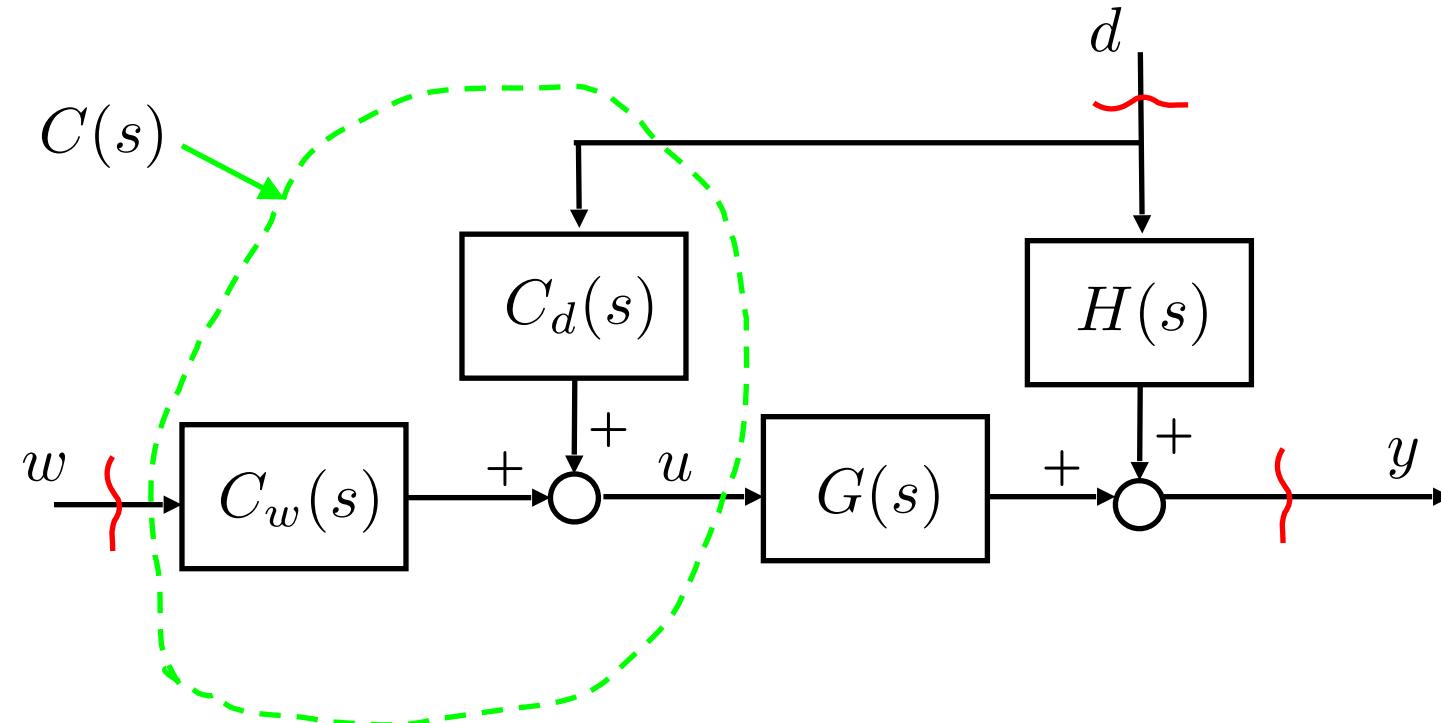
$$F_1(s) = \frac{1}{1 + 0.01s}$$

$$F_2(s) = \frac{1}{1 + 0.1s}$$

$$F_3(s) = \frac{1 + s}{(1 + 2s)(1 + 0.1s)}$$



# Open-Loop Control Systems with Disturbance Rejection



**Assumption:** the disturbance  $d(t)$  is **accessible** for measurement

# Open-Loop Control Systems – “Ideal” Performance



$$y(t) = w(t), t \geq 0 \quad \rightarrow \quad \frac{Y(s)}{W(s)} = 1 \quad \rightarrow \quad C_w(s)G(s) = 1$$

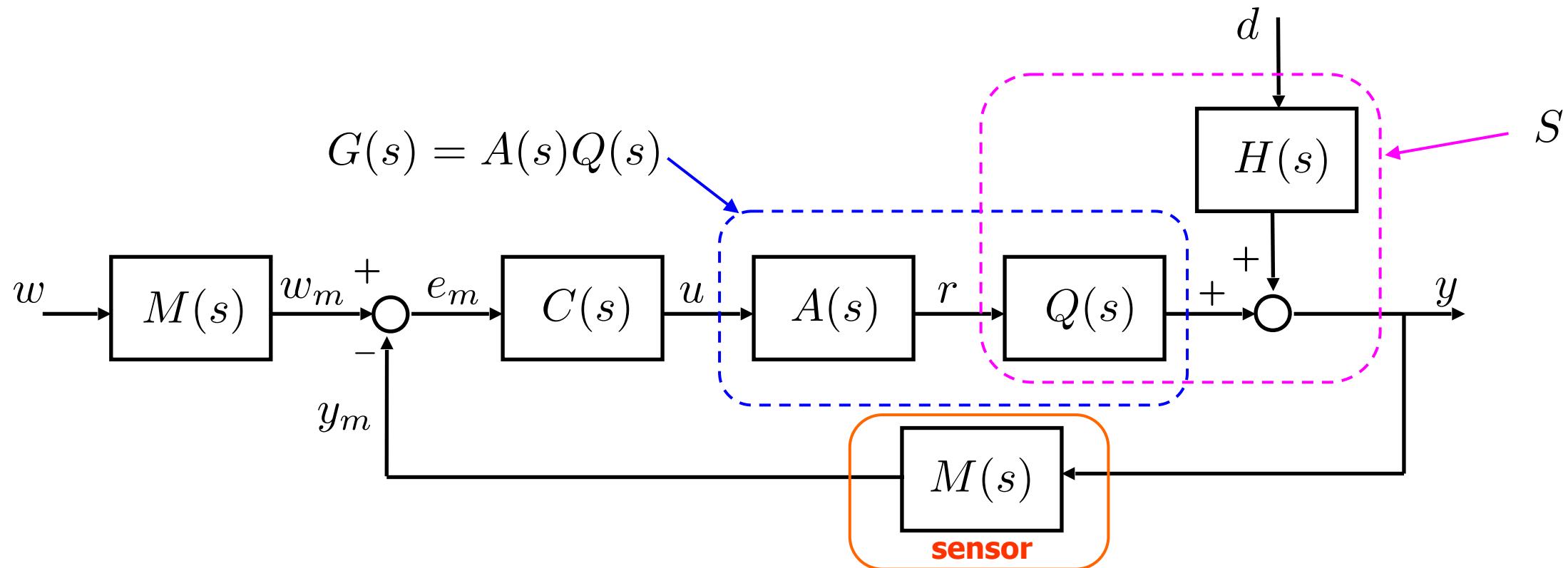
↳  $C_w(s) = G(s)^{-1}$

$$y(t) = 0, t \geq 0 \text{ if } w(t) = 0, t \geq 0 \quad \rightarrow \quad \frac{Y(s)}{D(s)} = 0 = H(s) + C_d(s) \cdot G(s)$$

↳  $C_d(s) = -G(s)^{-1}H(s)$

Hence, the **same limitations** we have seen in the no-disturbance case apply here as well because of the need of “inverting” the system’s dynamics

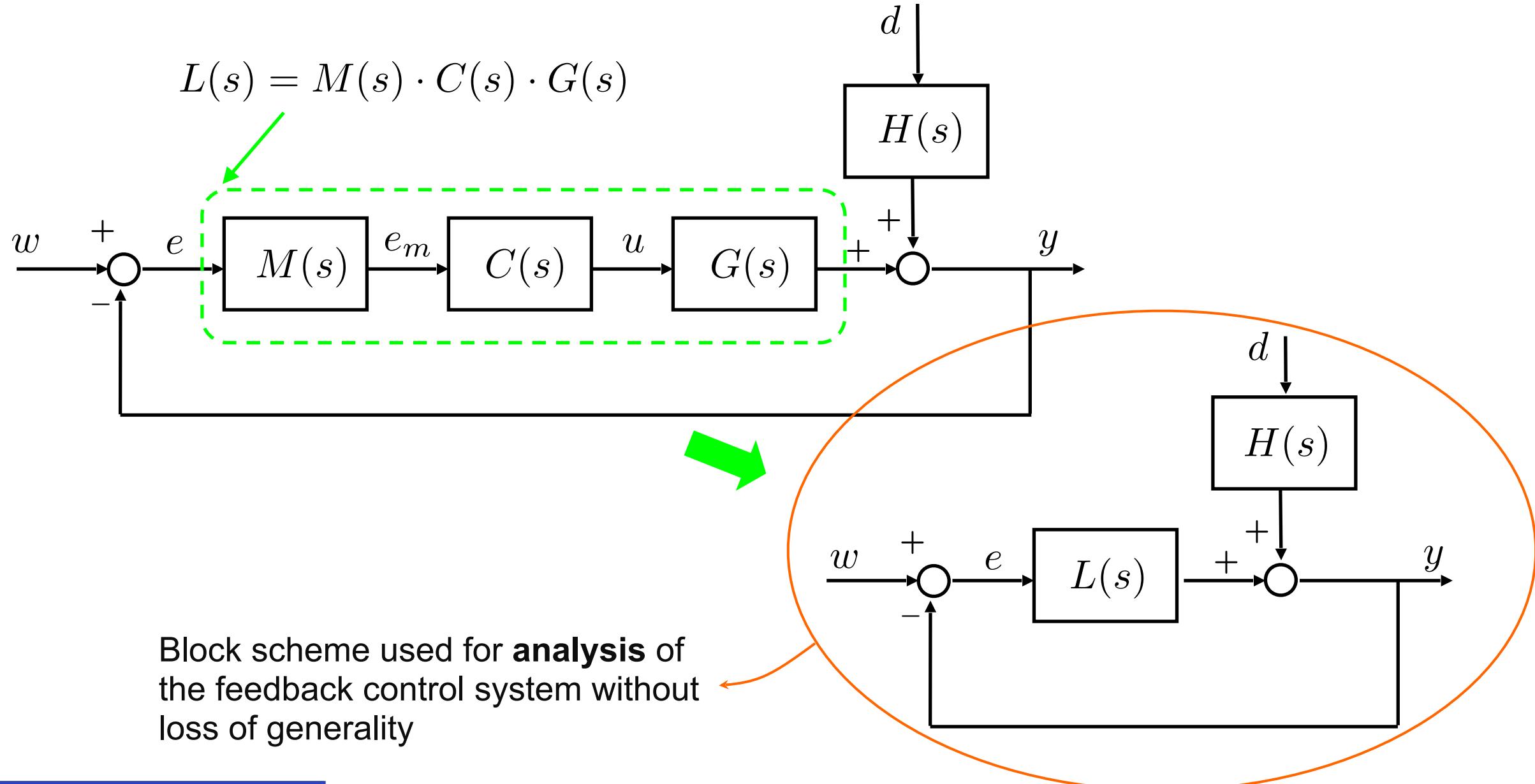
# Feedback (Closed-Loop) Control Systems



↳  $E_m(s) = W_m(s) - Y_m(s)$

$$= M(s)[W(s) - Y(s)] = M(s) \cdot E(s)$$

# Feedback (Closed-Loop) Control Systems (contd.)



# **Analysis of Feedback Control Systems**

## **Methods and Tools in the Frequency Domain**

# Feedback Control Systems – “Ideal” Performance

$$y(t) = w(t), t \geq 0 \quad \longrightarrow \quad F(s) = \frac{Y(s)}{W(s)} = 1$$

$$y(t) = 0, t \geq 0 \text{ if } w(t) = 0, t \geq 0 \quad \longrightarrow \quad R(s) = \frac{Y(s)}{D(s)} = 0$$

However, in general:

$$F(s) = \frac{L(s)}{1 + L(s)} \neq 1$$

$$R(s) = \frac{H(s)}{1 + L(s)} \neq 0$$

The **realistic scenario to be achieved** is:

- $F(s)$  low-pass filter with a sufficiently large bandwidth and gain  $\mu_F = 1$
- $|R(j\omega)| \simeq 0$  in the frequency range where the spectrum of  $d(t)$  is significant

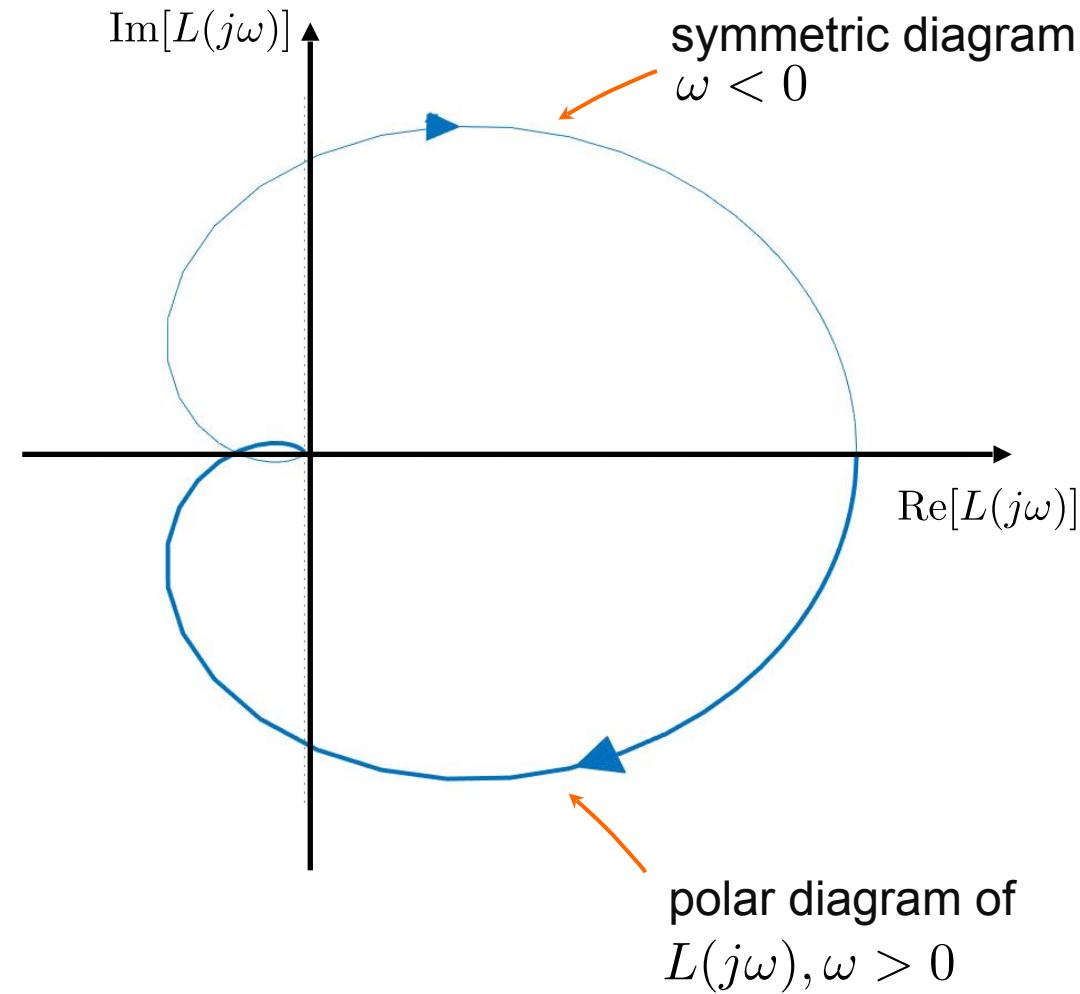
# Nyquist Diagram



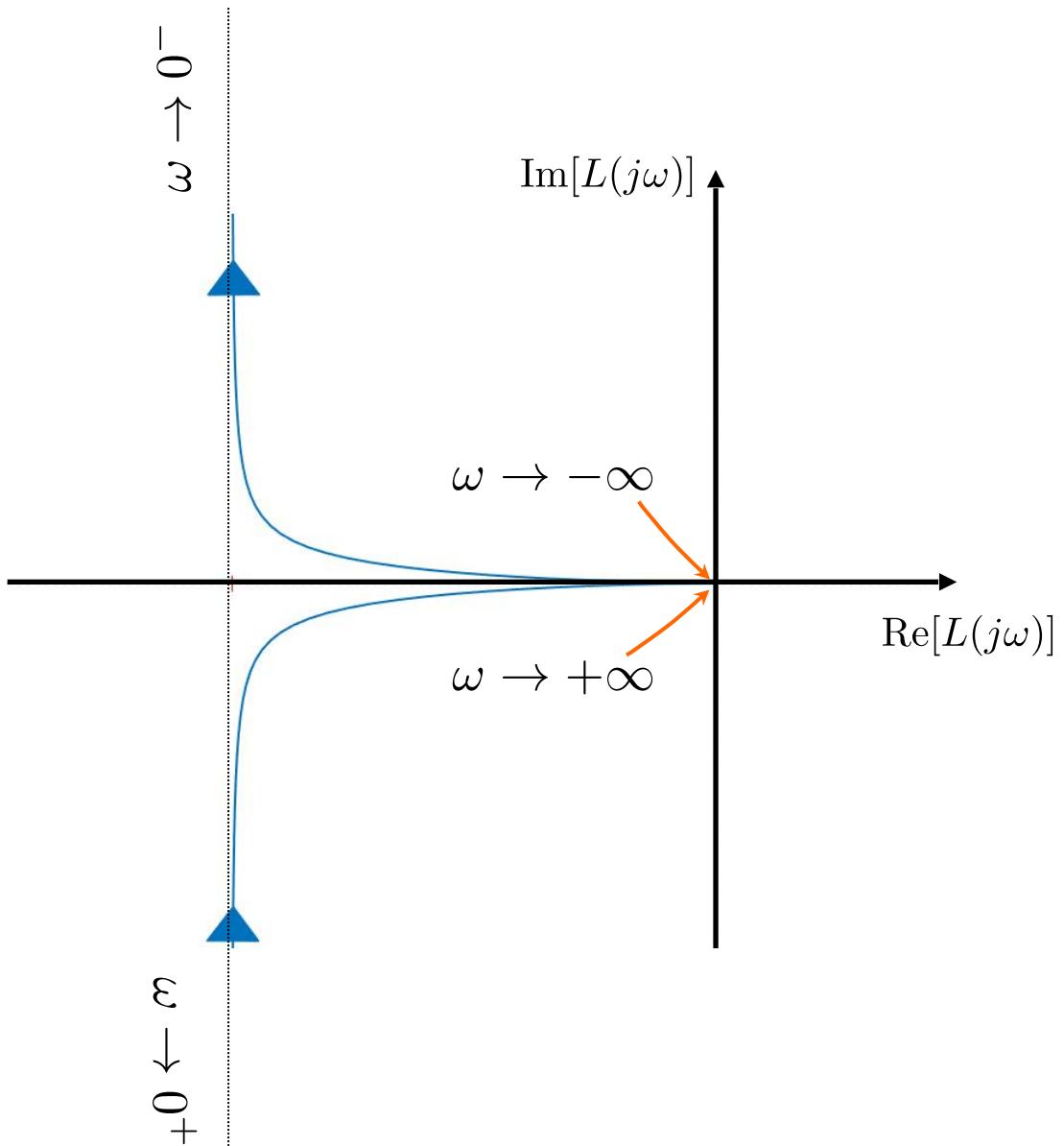
- The **Nyquist diagram**  $\Gamma$  is an extension of the conventional polar diagram to the angular frequencies range  $-\infty < \omega < +\infty$ .
- The following property holds:

$$L(-j\omega) = L^*(j\omega)$$

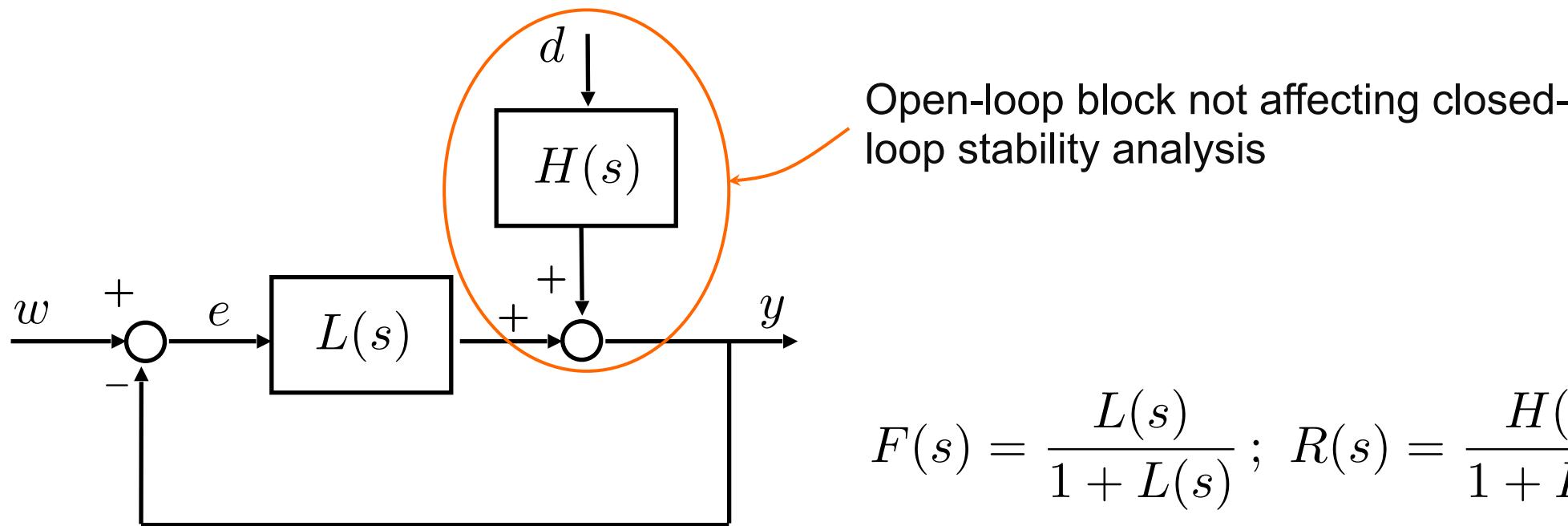
- The Nyquist diagram  $\Gamma$  is:
  - polar diagram of  $L(j\omega)$ ,  $\omega \geq 0$
  - +
  - the diagram symmetric to the polar diagram with respect to the real axis



If the polar diagram of  $L(j\omega)$ ,  $\omega \geq 0$  is not closed and bounded a **clock-wise “infinite-width” closure diagram** must be added to obtain the Nyquist diagram  $\Gamma$



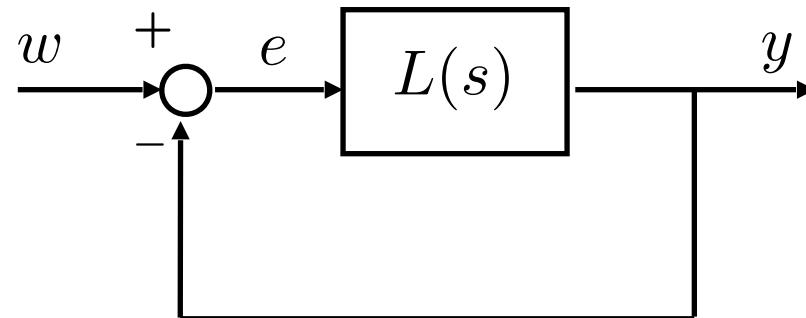
# Nyquist Closed-Loop Stability Criterion



- Closed-loop stability analysis boils down to establishing the location of the **zeros** of  $1 + L(s)$  in the complex plane, that is, analysing the solutions of

$$1 + L(s) = 0$$

## Nyquist Closed-Loop Stability Criterion (contd.)



$$1 + L(s) = 0$$

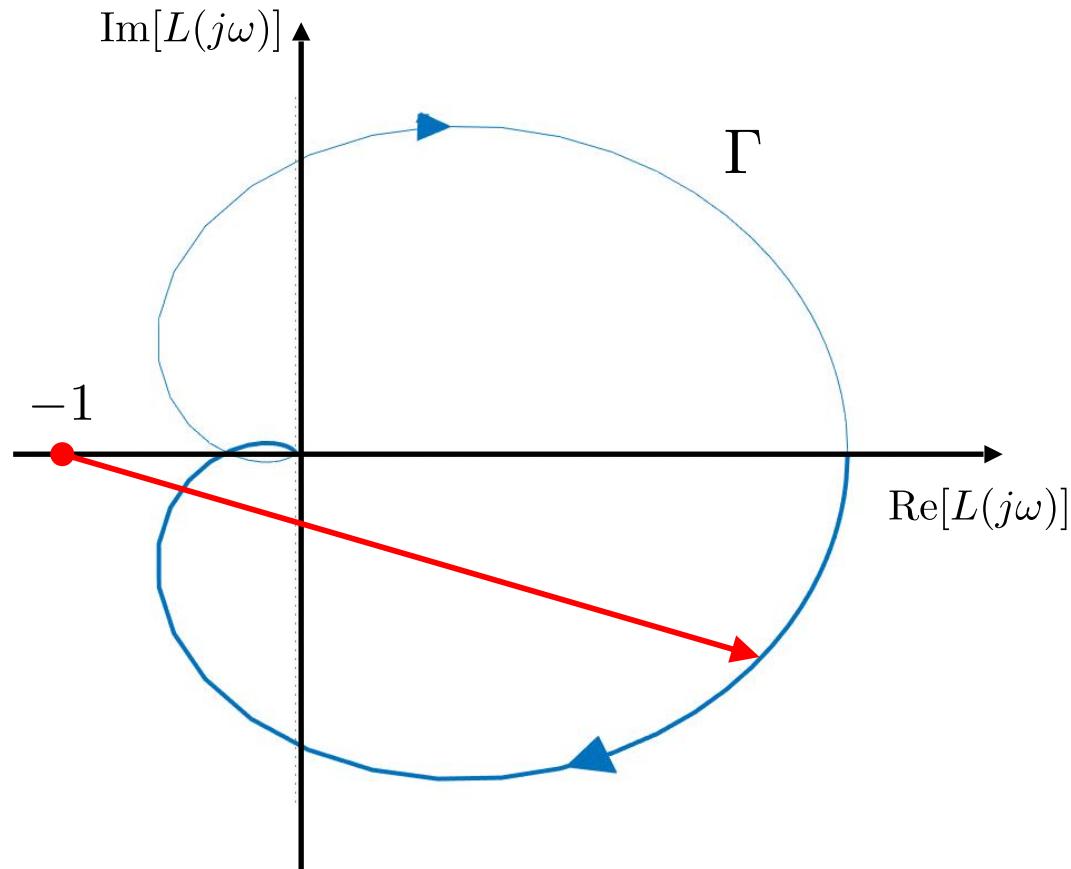
- Nyquist diagram  $\Gamma$  of  $L(s)$
- Number  $N$  of **counterclockwise** rotations of  $\Gamma$  around point  $(-1, 0) \in \mathbb{C}$
- Number  $n_{p>0}$  of poles of  $L(s)$  in the right half-plane

Closed-loop Asymptotic Stability



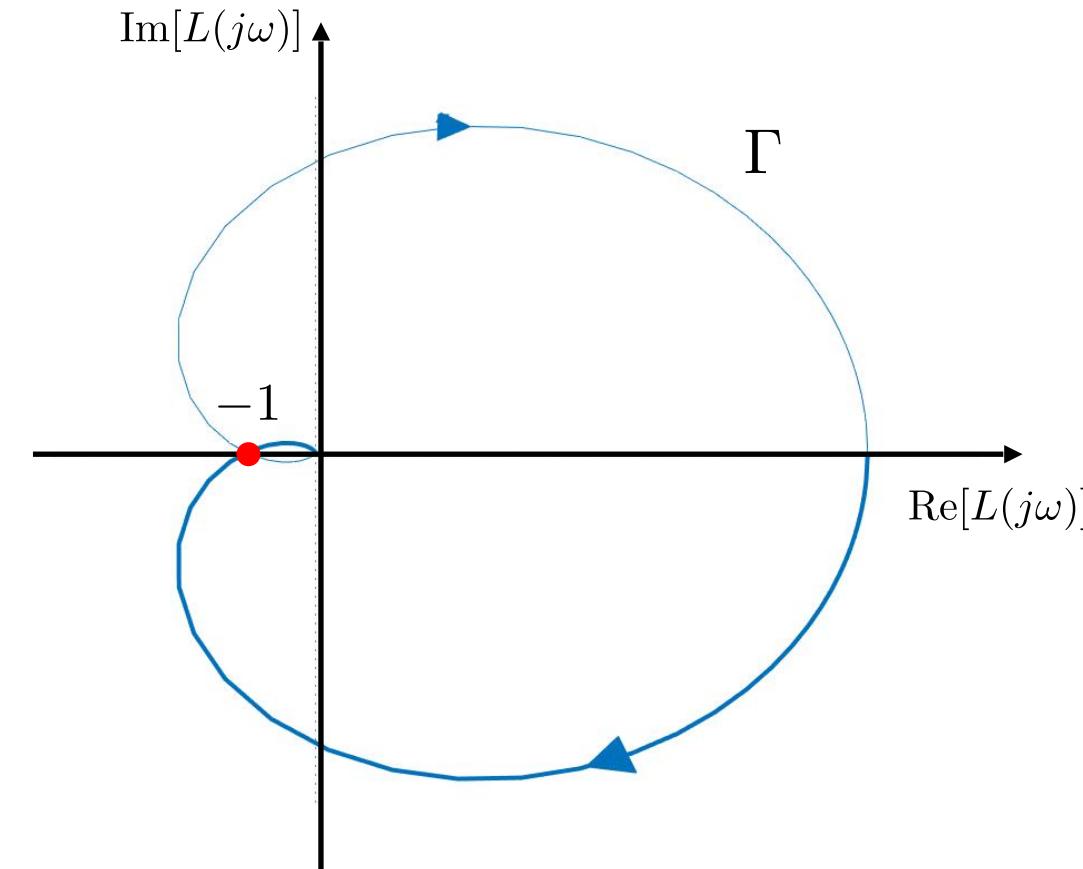
- $N$  "well-defined"
- $N = n_{p>0}$

## Remarks



The point  $(-1, 0)$  lies outside  
of the Nyquist diagram  $\Gamma$

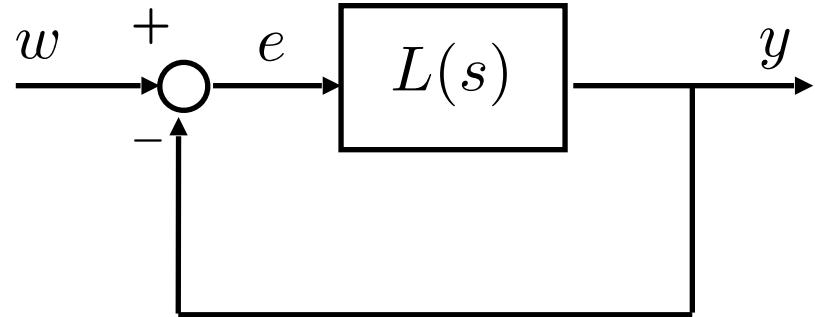
$$\begin{array}{l} \blacktriangleleft \\ N = 0 \end{array}$$



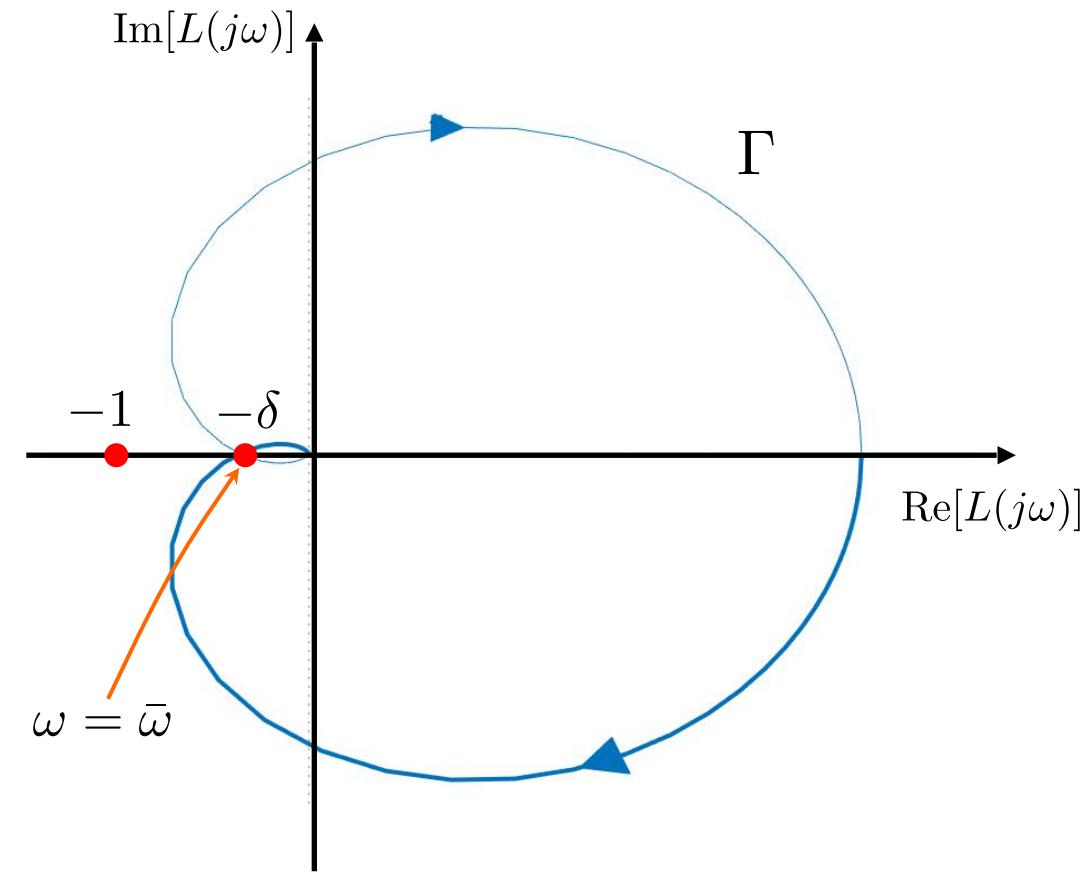
The point  $(-1, 0)$  lies on the  
Nyquist diagram  $\Gamma$

$$\begin{array}{l} \blacktriangleleft \\ N \text{ undefined} \end{array}$$

# Nyquist Closed-Loop Stability Criterion - Intuitive Justification

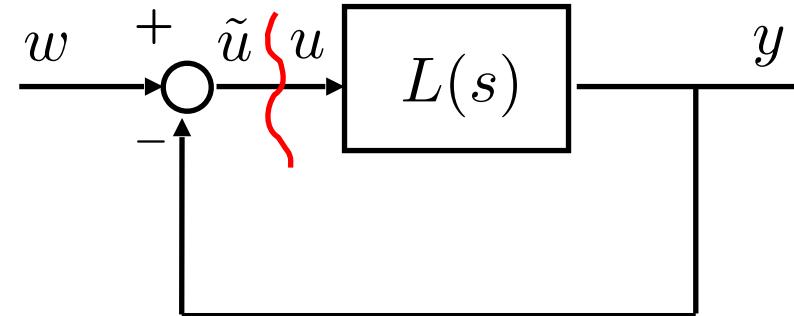


Suppose for simplicity  
that the open-loop  
system  $L(s)$  is  
asymptotically stable,  
hence  $n_{p>0} = 0$



$$L(j\bar{\omega}) = -\delta \quad \longleftrightarrow \quad |L(j\bar{\omega})| = \delta \quad \arg L(j\bar{\omega}) = -180^\circ$$

Hypothetically, suppose to “break the loop” in an arbitrary point:



$$u(t) = \sin(\bar{\omega}t)$$

↳  $y(t) \simeq |L(j\bar{\omega})| \sin[\bar{\omega}t + \arg(L(j\bar{\omega}))]$

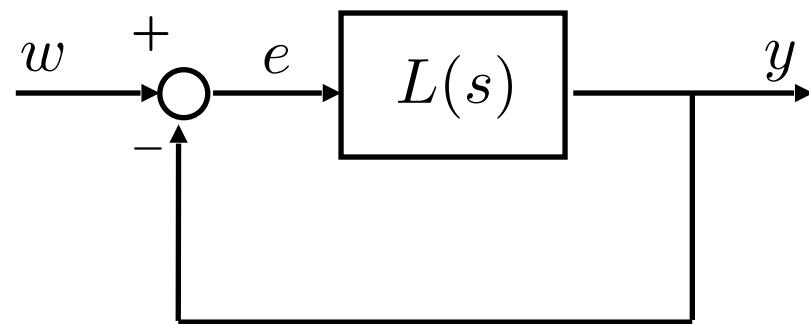
$$= \delta \sin(\bar{\omega}t - \pi) = -\delta \sin(\bar{\omega}t)$$

↳  $\tilde{u}(t) = \delta \sin(\bar{\omega}t)$

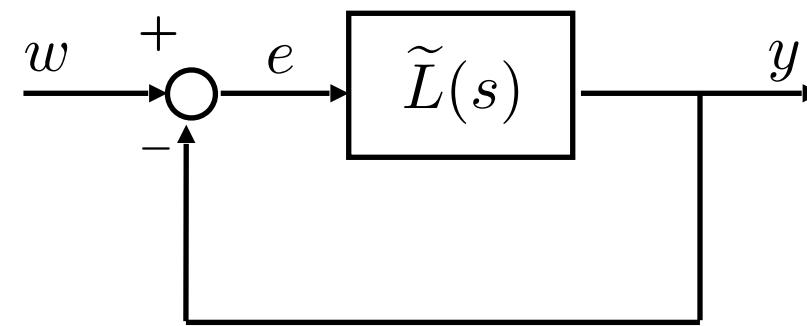
Hence:

- $\delta > 1$  → unstable [  $N \neq 0 (= n_{p>0})$  ]
- $\delta < 1$  → asymptotically stable [  $N = 0 (= n_{p>0})$  ]
- $\delta = 1$  → not asymptotically stable [  $N$  undefined ]

# Closed-Loop Robust Stability



nominal model

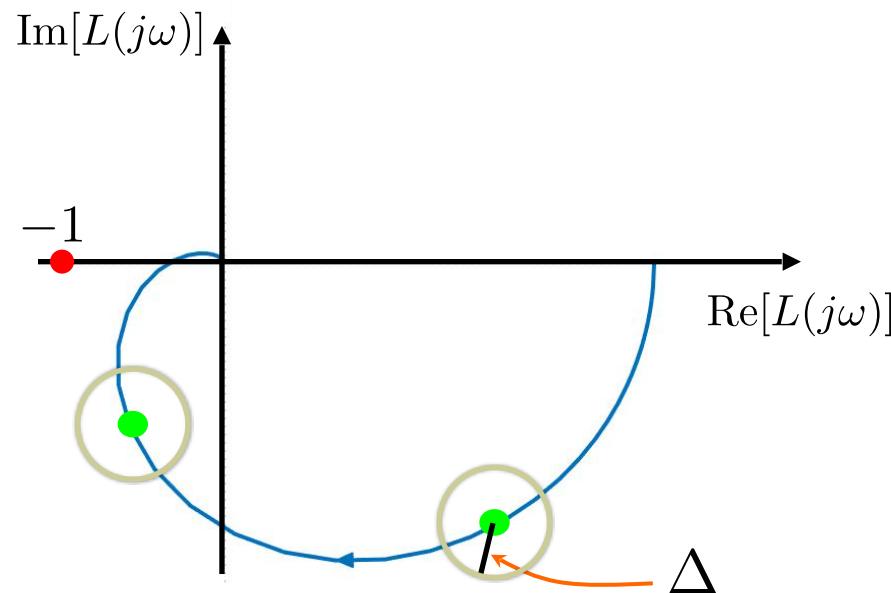


true model

- In practical engineering contexts  $\tilde{L}(s) \neq L(s)$
- The aim is guaranteeing the **closed-loop robust stability**, that is, the **stability in the presence of uncertainty on the open-loop nominal model  $L(s)$**
- A **mathematical characterisation of the uncertainty** is needed

- **Unstructured Uncertainty:**

$$\tilde{L}(s) = L(s) + \delta L(s); |\delta L(j\omega)| \leq \Delta$$



- **Open-Loop Gain Uncertainty**

$$\tilde{L}(s) = K \cdot L(s); 0 < K < \bar{K}$$

- We suppose that the **nominal** open-loop model  $L(s)$  is asymptotically stable
- Hence, to guarantee closed-loop asymptotic stability for the nominal model, the nominal Nyquist diagram must **not encircle the point  $(-1, 0)$**
- **Robust stability indicators** quantify the magnitude of the uncertainty for which closed-asymptotic stability is preserved when the nominal model is **replaced with the true model**  $\tilde{L}(s)$  in the closed-loop scheme
- Robust stability indicators also quantify the “**distance**” of the nominal closed-loop system **from the "instability scenario"**

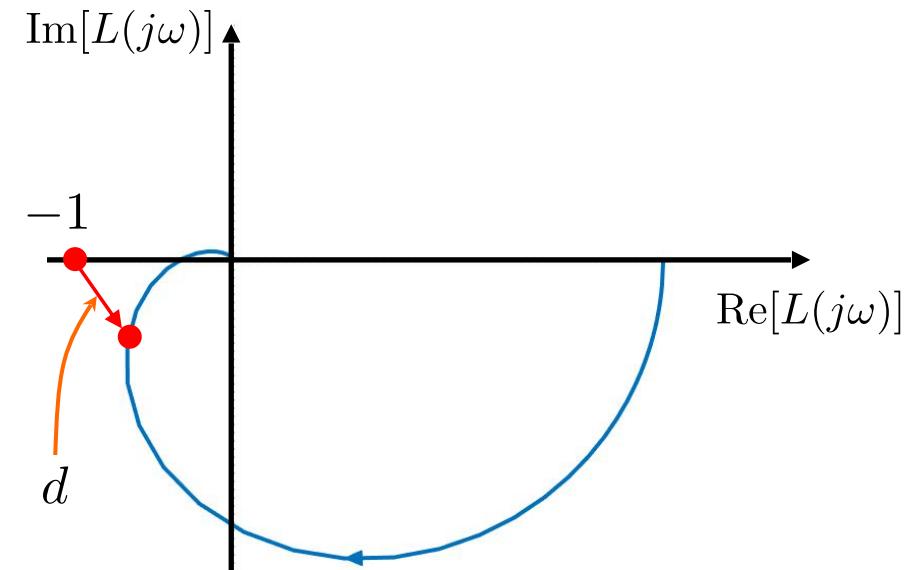
# A Candidate Robust Stability Indicator



The more natural choice as robust stability indicator is the Euclidean distance of the polar diagram  $L(j\omega)$  from the point  $(-1, 0)$

**vector stability margin:**

$$d = \min_{\omega} |1 + L(j\omega)|$$

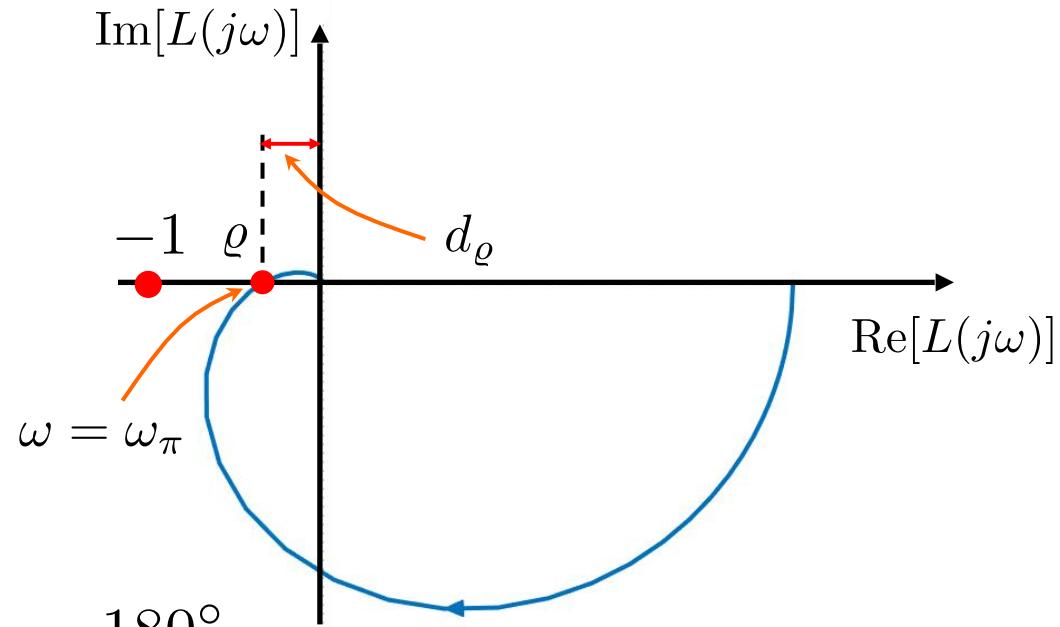


The vector stability margin is well-defined mathematically, but it is **not very useful in practice** since it cannot be evaluated using the Bode diagrams of  $L(j\omega)$

The **gain margin** is a robust stability indicator defined as:

**gain margin:**

$$K_m = \frac{1}{d_\varrho}$$



Hence:

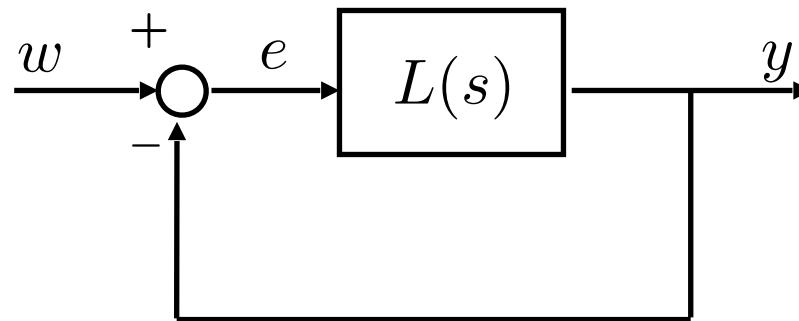
$$\omega_\pi : \omega \text{ such that } \arg L(j\omega) = -180^\circ$$



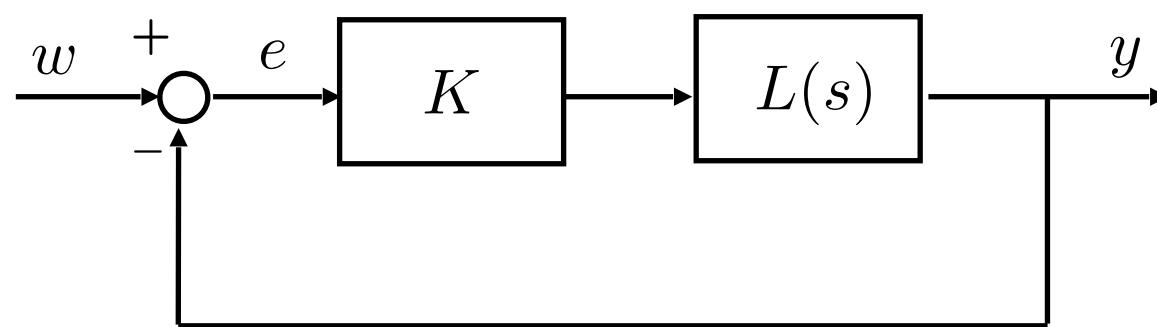
$$K_m = \frac{1}{|L(j\omega_\pi)|} = -|L(j\omega_\pi)|_{\text{dB}}$$

$K_m$  can be evaluated from the Bode diagrams

# Gain Margin - Interpretation

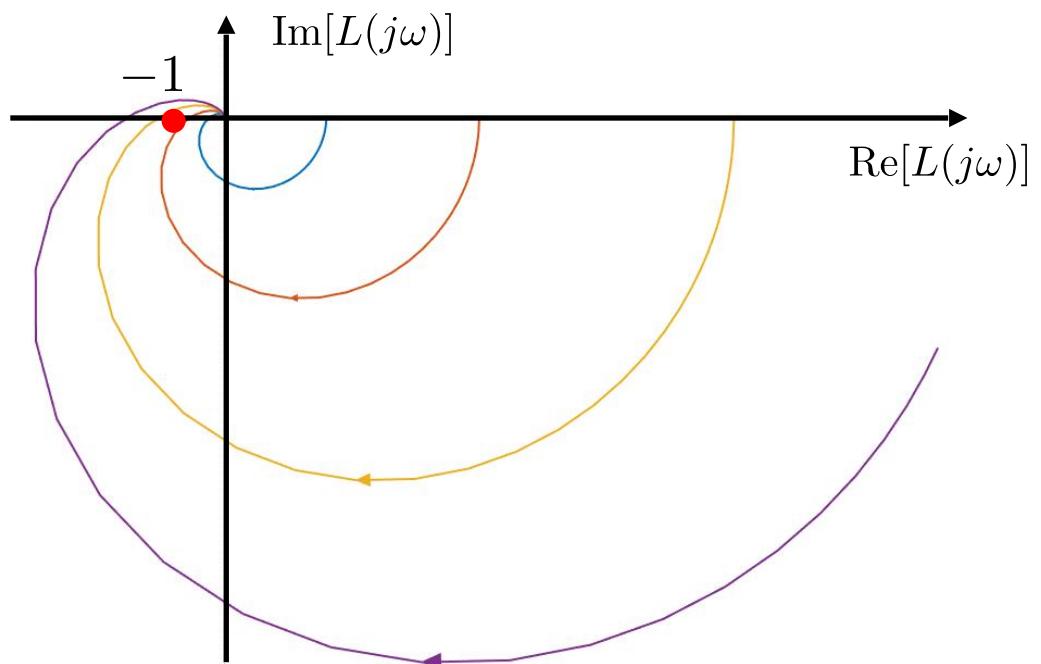


nominal model



$0 < K < K_m \rightarrow$  asymptotically stable

The gain margin  $K_m$  is a robustness indicator referring to **uncertainty on the loop gain**



The **phase margin** is a robust stability indicator defined as:

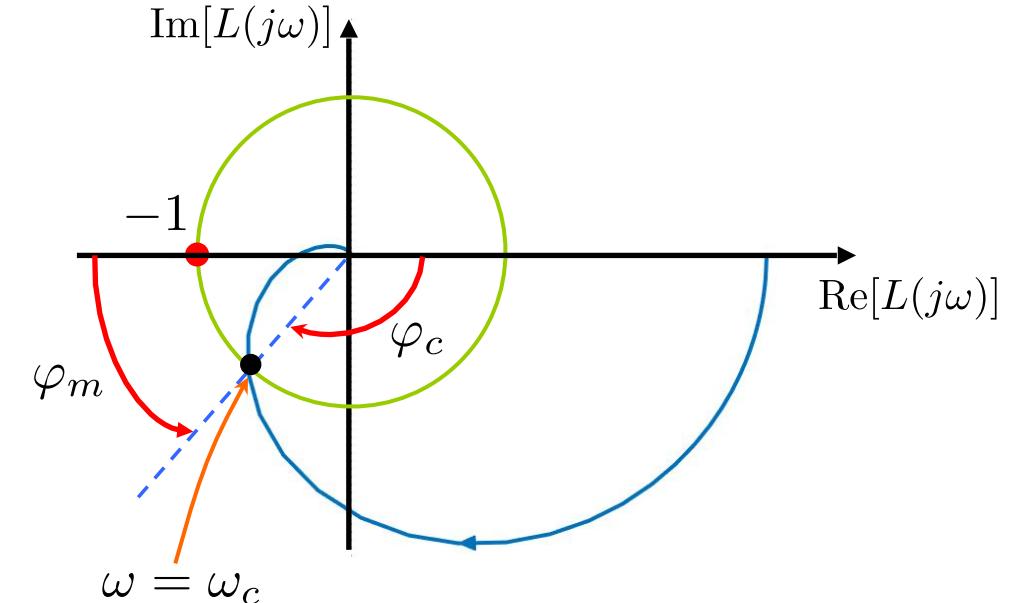
**phase margin:**

$$\varphi_m = \varphi_c - (-180^\circ)$$

Remark: recall that  $\varphi_c < 0$  in the figure on the left consistently with the conventions we have used in the Nyquist and Bode diagrams (see Part 8)

Hence:

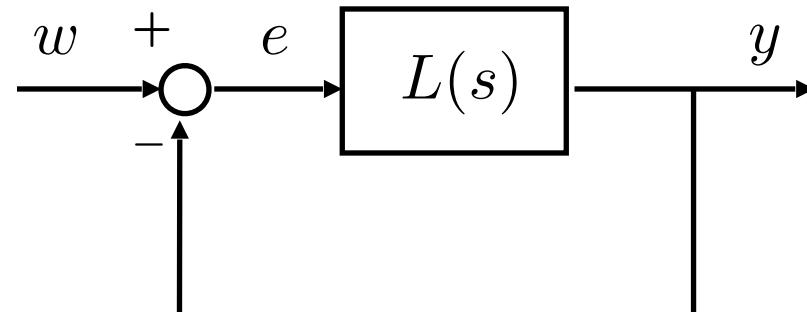
$$\omega_c : \omega \text{ such that } |L(j\omega_c)| = 1 = 0\text{dB}$$



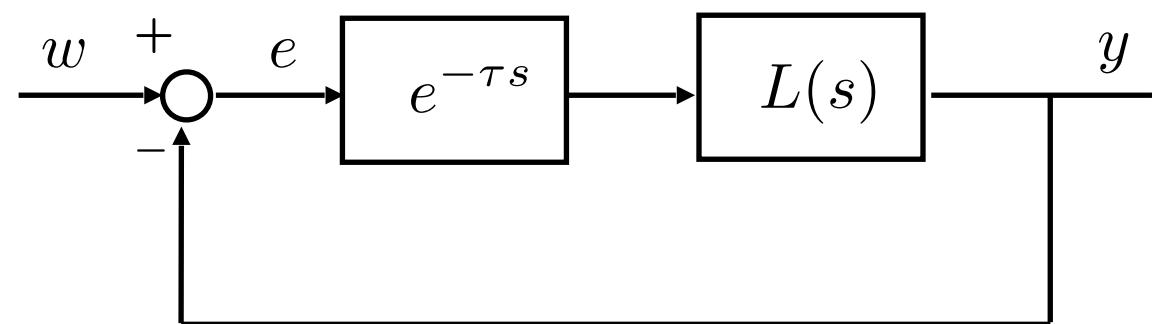
$$\varphi_c = \arg L(j\omega_c)$$

$\varphi_m = \varphi_c - (-180^\circ)$  can be evaluated from the Bode diagrams

# Phase Margin - Interpretation

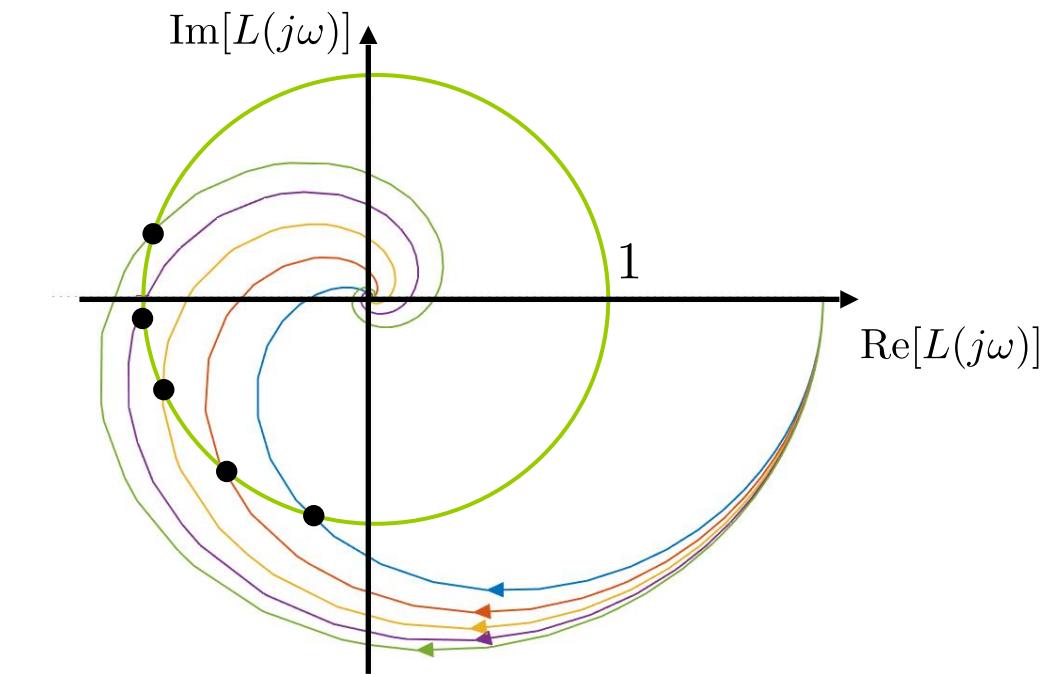


nominal model



Since  $\omega_c \tau = \varphi_m \cdot \frac{\pi}{180}$   
↳  $0 < \tau < \frac{\varphi_m}{\omega_c} \cdot \frac{\pi}{180}$   
asymptotically stable

The phase margin  $\varphi_m$  is a robustness indicator referring to **uncertainty on the loop "delay"**



# Gain and Phase Margin – Evaluation

- **gain margin:**

$$K_m = \frac{1}{d_\varrho}$$

↳  $K_m = \frac{1}{|L(j\omega_\pi)|} = -|L(j\omega_\pi)|_{\text{dB}}$

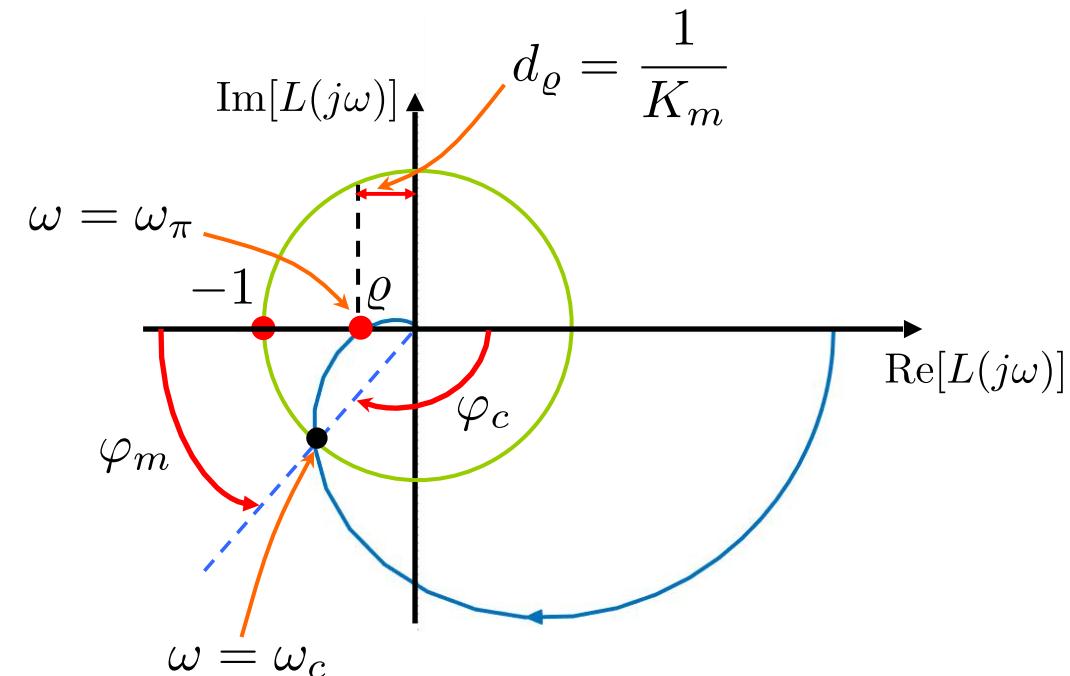
$$\arg L(j\omega_\pi) = -180^\circ$$

- **phase margin:**

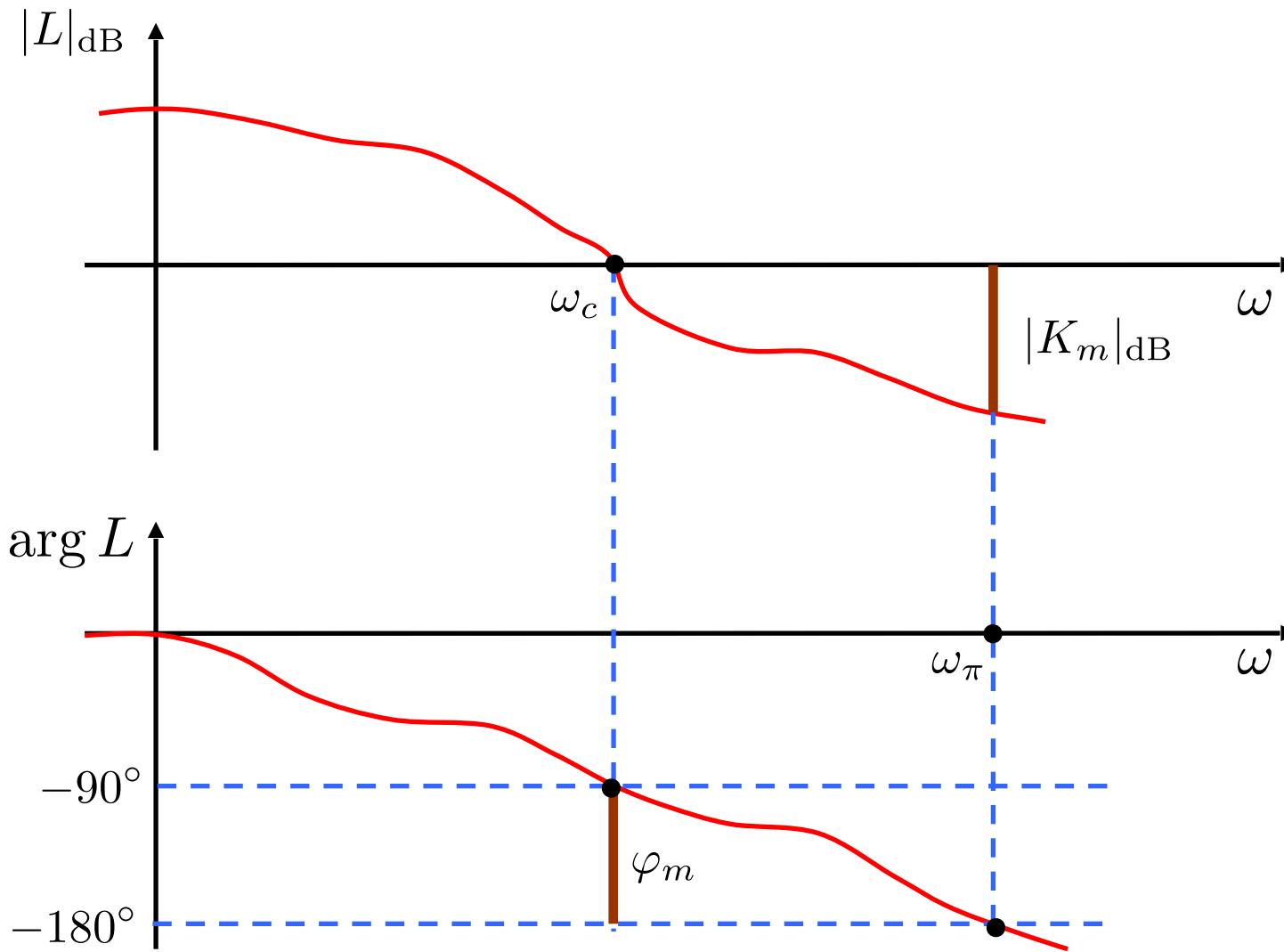
$$\varphi_m = \varphi_c - (-180^\circ)$$

↳  $\omega_c$  :  $\omega$  such that  $|L(j\omega_c)| = 1 = 0\text{dB}$

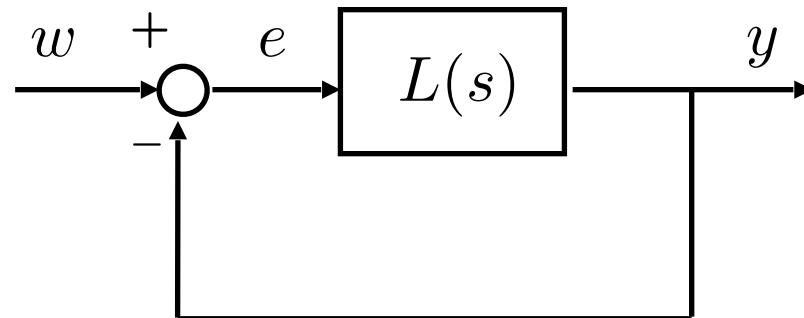
↳  $\varphi_c = \arg L(j\omega_c)$



# Gain and Phase Margin – Evaluation from Bode Diagrams



The Bode stability criterion is very useful for controller design even if it is applicable in a more restricted scenario than the Nyquist stability criterion:



- Loop gain:  $\mu$
- Phase Margin:  $\varphi_m$

## Assumptions:

- Open-loop asymptotic stability:  $n_{p>0} = 0$
- The magnitude Bode diagram  $|L(j\omega)|_{\text{dB}}$  crosses the 0dB axis "only one time from top to bottom"

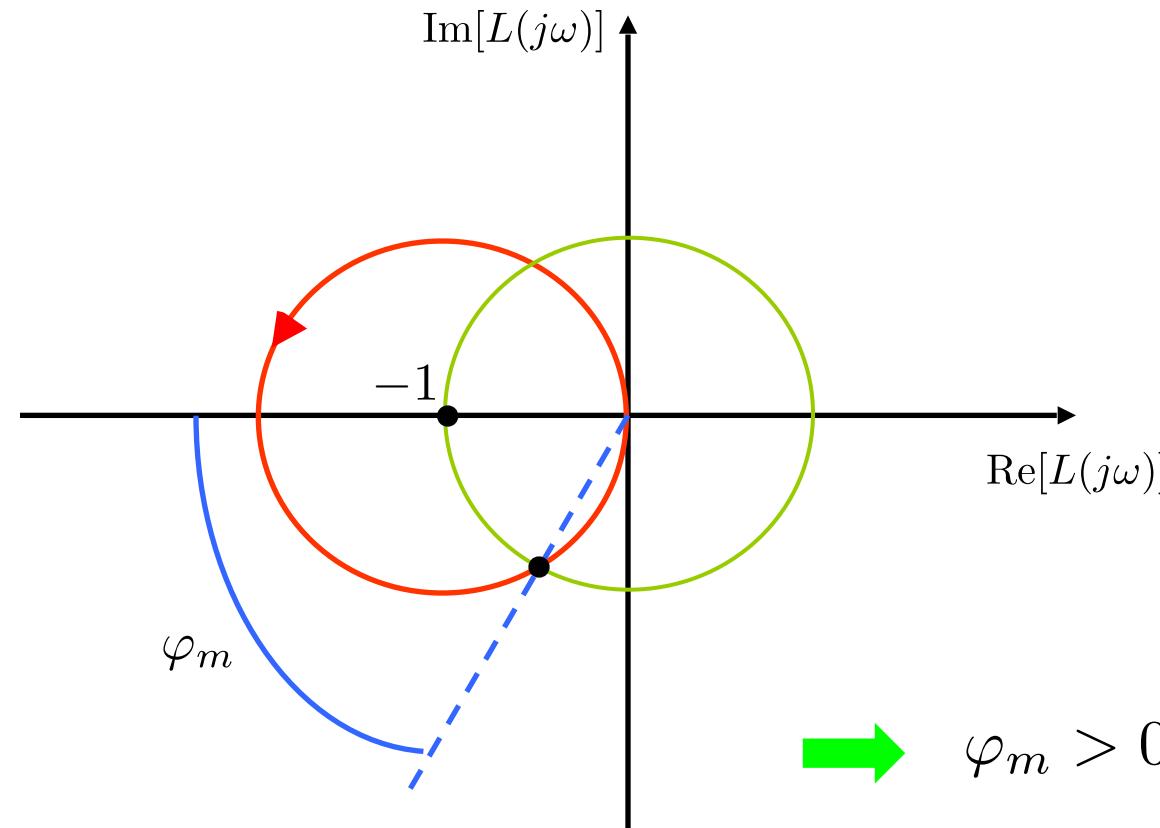


Closed-loop Asymptotic Stability



- $\mu > 0$
- $\varphi_m > 0$

The assumption on **positivity of the loop-gain**  $\mu > 0$  is important to avoid cases like:



$\varphi_m > 0$  **BUT** closed-loop unstable



Recall from Part 8, slide 19:

- **Minimum Phase Systems** are characterized by:
  - positive gain:  $\mu > 0$
  - all poles and zeros located in the left half-plane
- For minimum phase systems there is a **direct relation among the approximate Bode diagrams of magnitude and phase**:

	Slope of $ L(j\omega) _{\text{dB}}$	Value of $\arg L(j\omega)$
pole	-20dB/dec	$-90^\circ$
zero	20dB/dec	$90^\circ$

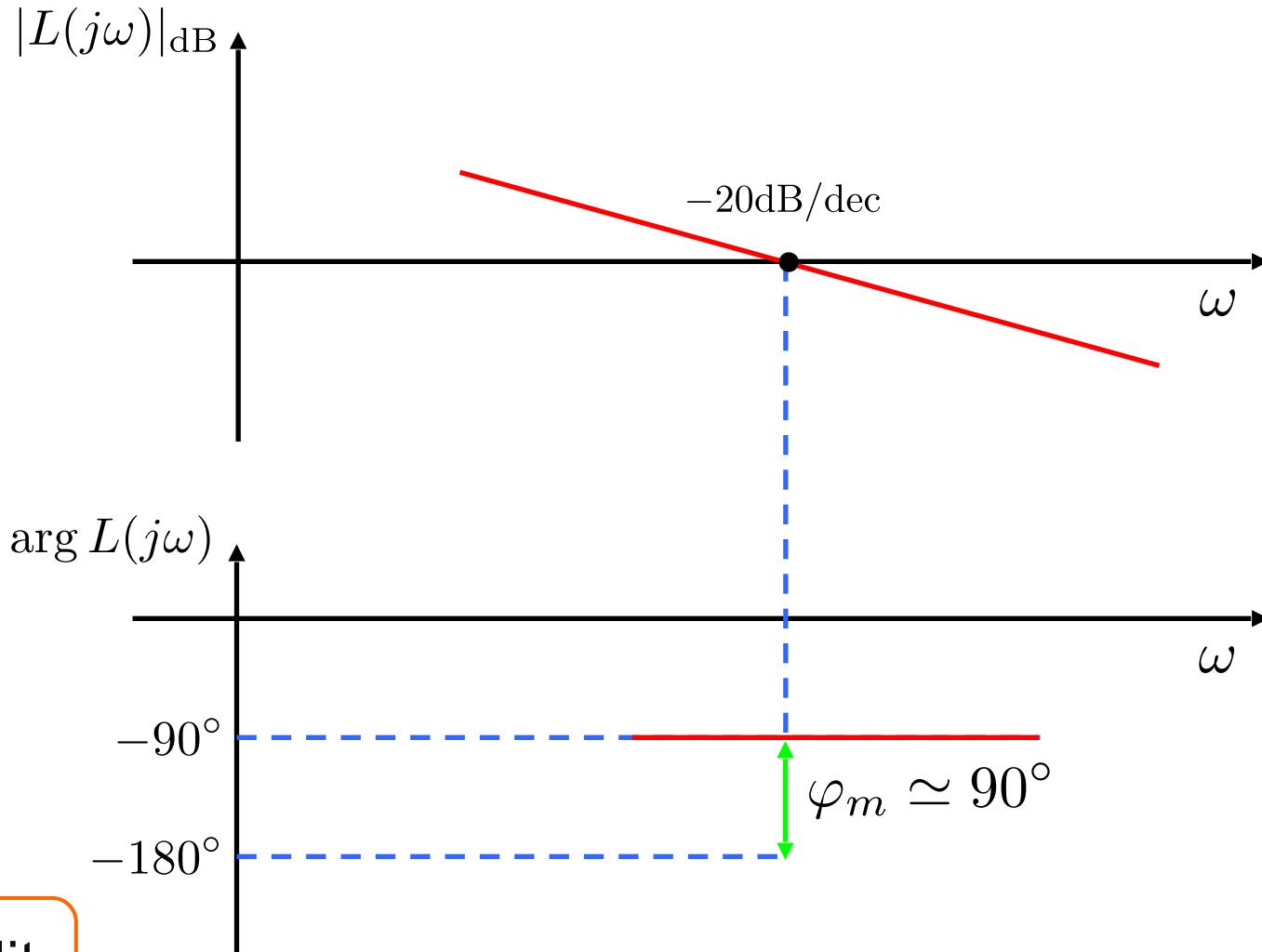
# A Useful Empirical Criterion for Minimum-Phase Systems (contd.)

If:

- The open-loop system is **minimum phase**
- The magnitude diagram  $|L(j\omega)|_{\text{dB}}$  **crosses the 0dB axis only one time from top to bottom**
- The **slope** of the asymptotic magnitude Bode diagram is **-20 dB/dec** in a **sufficiently large frequency range around the 0dB crossing**



Closed-loop Asymptotic Stability

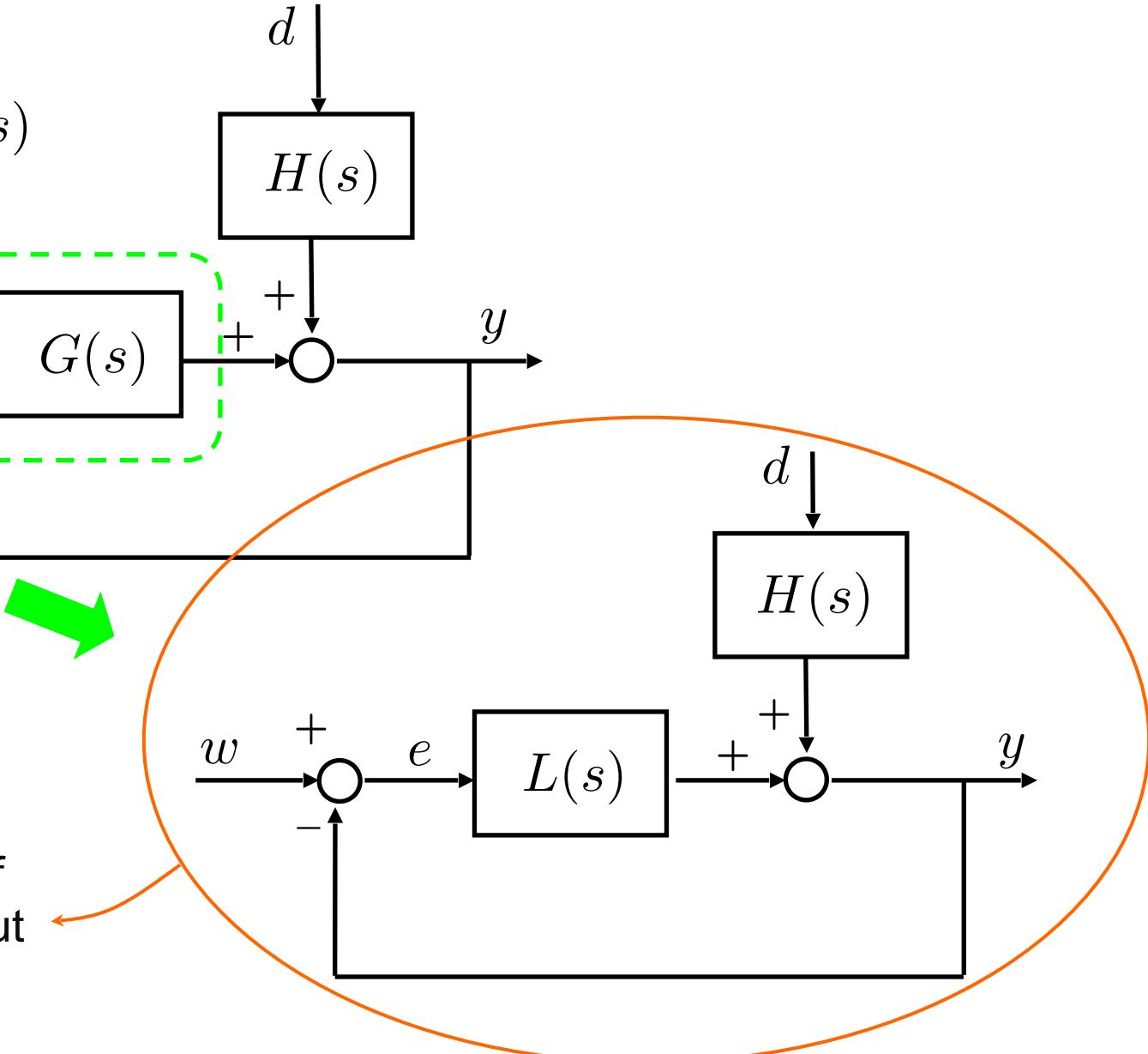
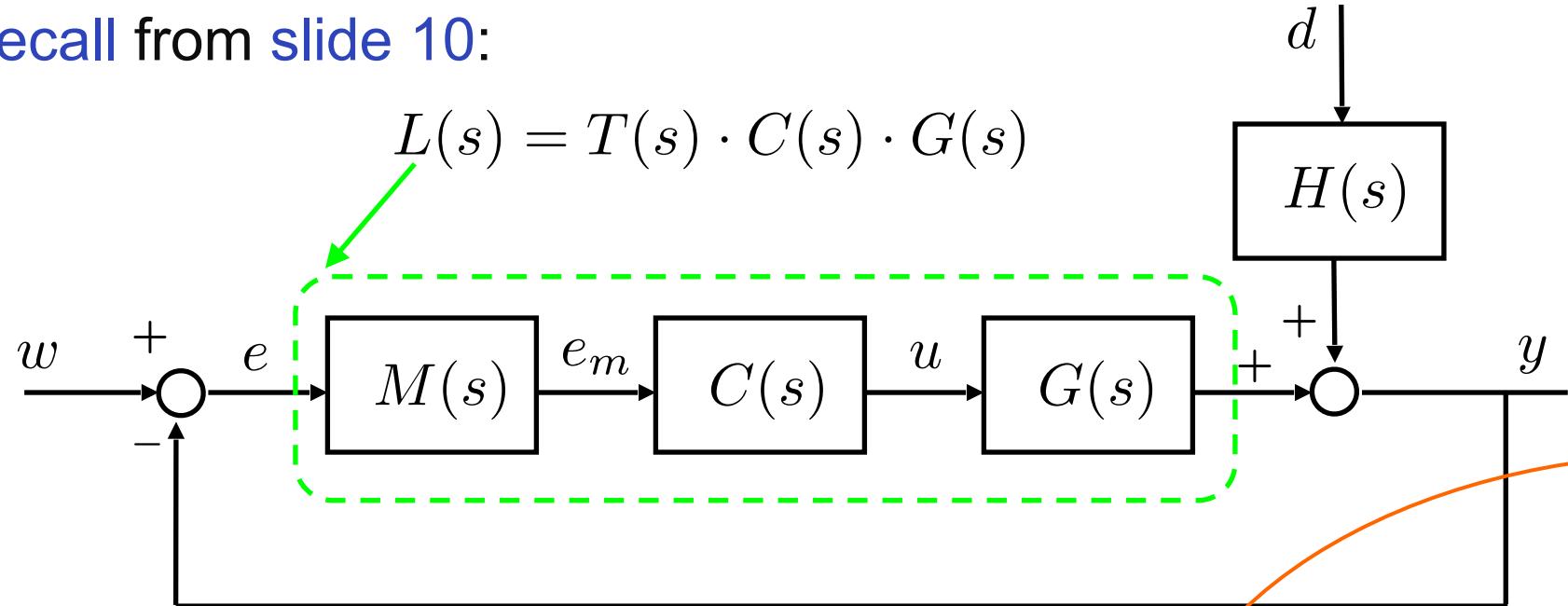


# Analysis of Feedback Control Systems



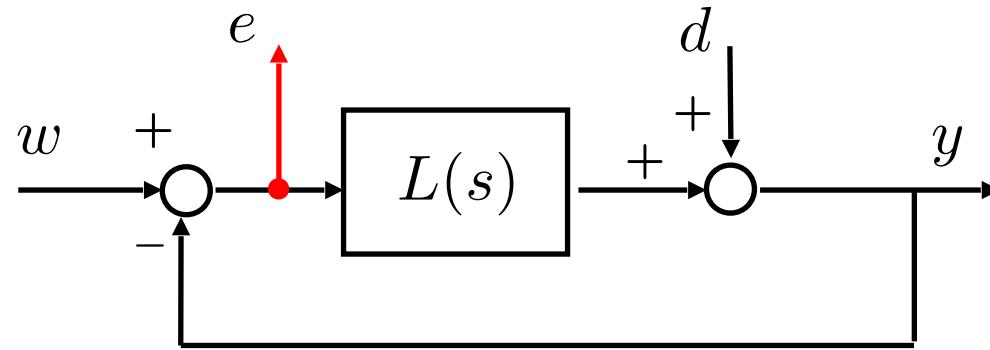
Recall from slide 10:

$$L(s) = T(s) \cdot C(s) \cdot G(s)$$



Block scheme used for **analysis** of  
the feedback control system without  
loss of generality

For the time being suppose that  $H(s) = 1$  :



Hence:

$$\frac{Y(s)}{W(s)} = \frac{L(s)}{1 + L(s)} = F(s); \quad \frac{Y(s)}{D(s)} = \frac{1}{1 + L(s)} = S(s)$$

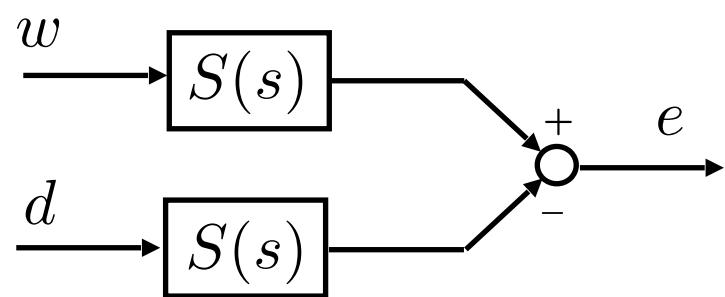
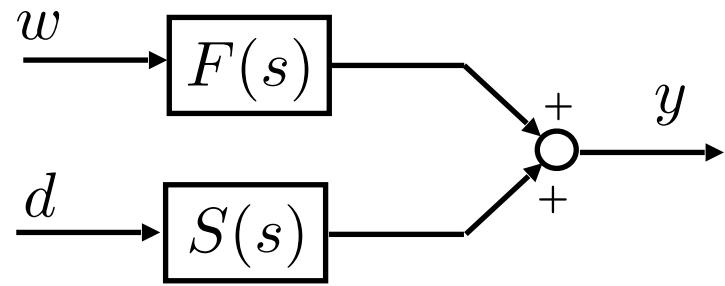
$$\frac{E(s)}{W(s)} = \frac{1}{1 + L(s)} = S(s); \quad \frac{E(s)}{D(s)} = \frac{-1}{1 + L(s)} = -S(s)$$

Define:

- **Complementary Sensitivity Function:**  $F(s)$
- **Sensitivity Function:**  $S(s)$

# Analysis of Feedback Control Systems (contd.)

Hence:



**Ideal Performance**

$$F(s) = \frac{L(s)}{1 + L(s)} \underset{\sim}{=} 1$$
$$S(s) = \frac{1}{1 + L(s)} \underset{\sim}{=} 0$$

Moreover:

$$F(s) + S(s) = 1$$

# Static Analysis of Complementary Sensitivity Function



Using the parametrisation of the open-loop t.f.  $L(s)$  in slides 32-33 Part IV:

$$L(s) = \mu \frac{1}{s^g} \frac{\prod_l \left(1 + \frac{s}{z_l}\right) \prod_h \left(1 + \frac{2\zeta_h}{\alpha_{nh}}s + \frac{1}{\alpha_{nh}^2}s^2\right)}{\prod_i \left(1 + \frac{s}{p_i}\right) \prod_k \left(1 + \frac{2\xi_k}{\omega_{nk}}s + \frac{1}{\omega_{nk}^2}s^2\right)}$$

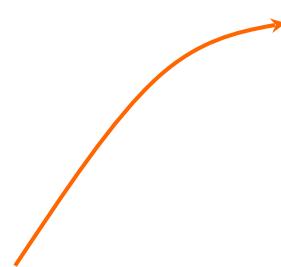
$s \rightarrow 0 \quad 1$

Asymptotic value of the **closed-loop step-response**:

$$\begin{aligned} w(t) &= A \cdot 1(t) \quad \rightarrow \quad y(\infty) &= \lim_{s \rightarrow 0} s \cdot F(s) \cdot \frac{A}{s} &= A \lim_{s \rightarrow 0} F(s) \\ && &= A \lim_{s \rightarrow 0} \frac{L(s)}{1 + L(s)} &= A \lim_{s \rightarrow 0} \frac{\frac{\mu}{s^g}}{1 + \frac{\mu}{s^g}} \\ && &= A \lim_{s \rightarrow 0} \frac{\mu}{s^g + \mu} \end{aligned}$$

Therefore:

$$y(\infty) = \begin{cases} A \cdot \frac{\mu}{1 + \mu} \text{ (hence } \mu_F \simeq 1 \text{ if } \mu \gg 1\text{)} & \text{if } g = 0 \\ A \text{ (hence } \mu_F = 1\text{)} & \text{if } g > 0 \\ 0 & \text{if } g < 0 \end{cases}$$



Presence of an integrator (pole = 0) in the direct path of the feedback loop

We have:

$$L(s) = \frac{N(s)}{\varphi(s)} \quad \rightarrow \quad F(s) = \frac{L(s)}{1 + L(s)} = \frac{N(s)}{\varphi(s) + N(s)}$$

Therefore:

- the zeros of  $F(s)$  coincide with the zeros of  $L(s)$
- the poles of  $F(s)$  coincide with the roots of  $\varphi(s) + N(s)$

# Frequency Response Analysis of Complementary Sensitivity Function

We have:

$$|F(j\omega)| = \frac{|L(j\omega)|}{|1 + L(j\omega)|} \simeq \begin{cases} 1 & \text{if } |L(j\omega)| \gg 1 \\ |L(j\omega)| & \text{if } |L(j\omega)| \ll 1 \end{cases}$$

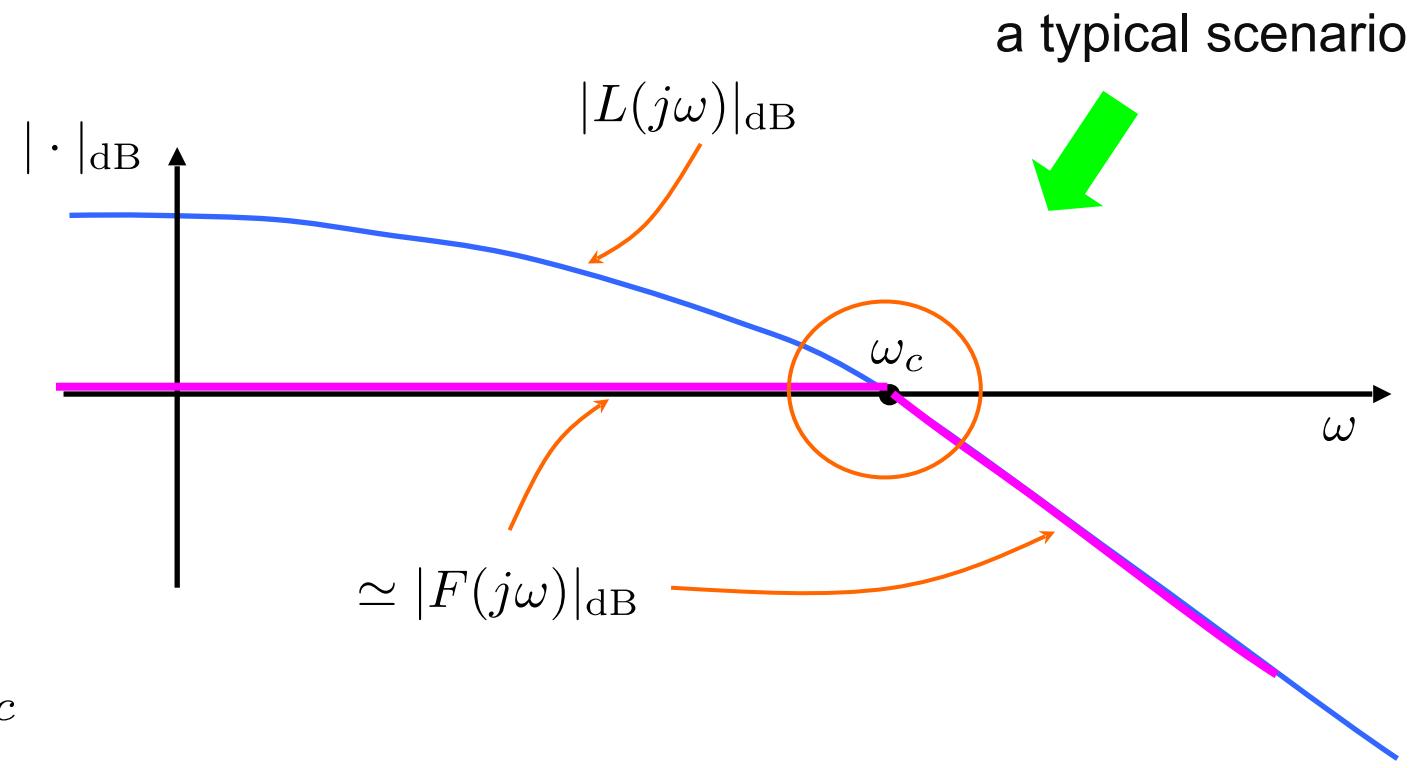
- $F(s)$  **low-pass filter** with bandwidth  $B \simeq [0, \omega_c]$
- Gain:

$$\mu_F = \begin{cases} 1 & \text{if } g > 0 \\ \frac{\mu}{1 + \mu} & \text{if } g = 0 \end{cases}$$

- Dominant poles:

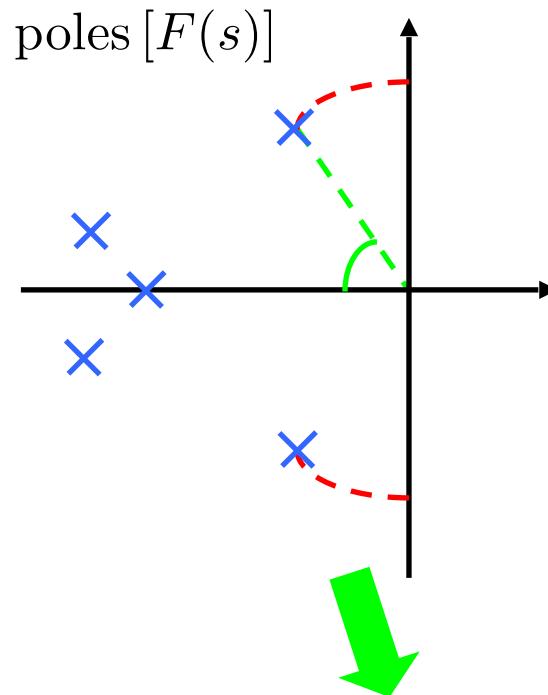
- if real:  $\rightarrow \tau \simeq \frac{1}{\omega_c}$

- if complex:  $\rightarrow \omega_n \simeq \omega_c$

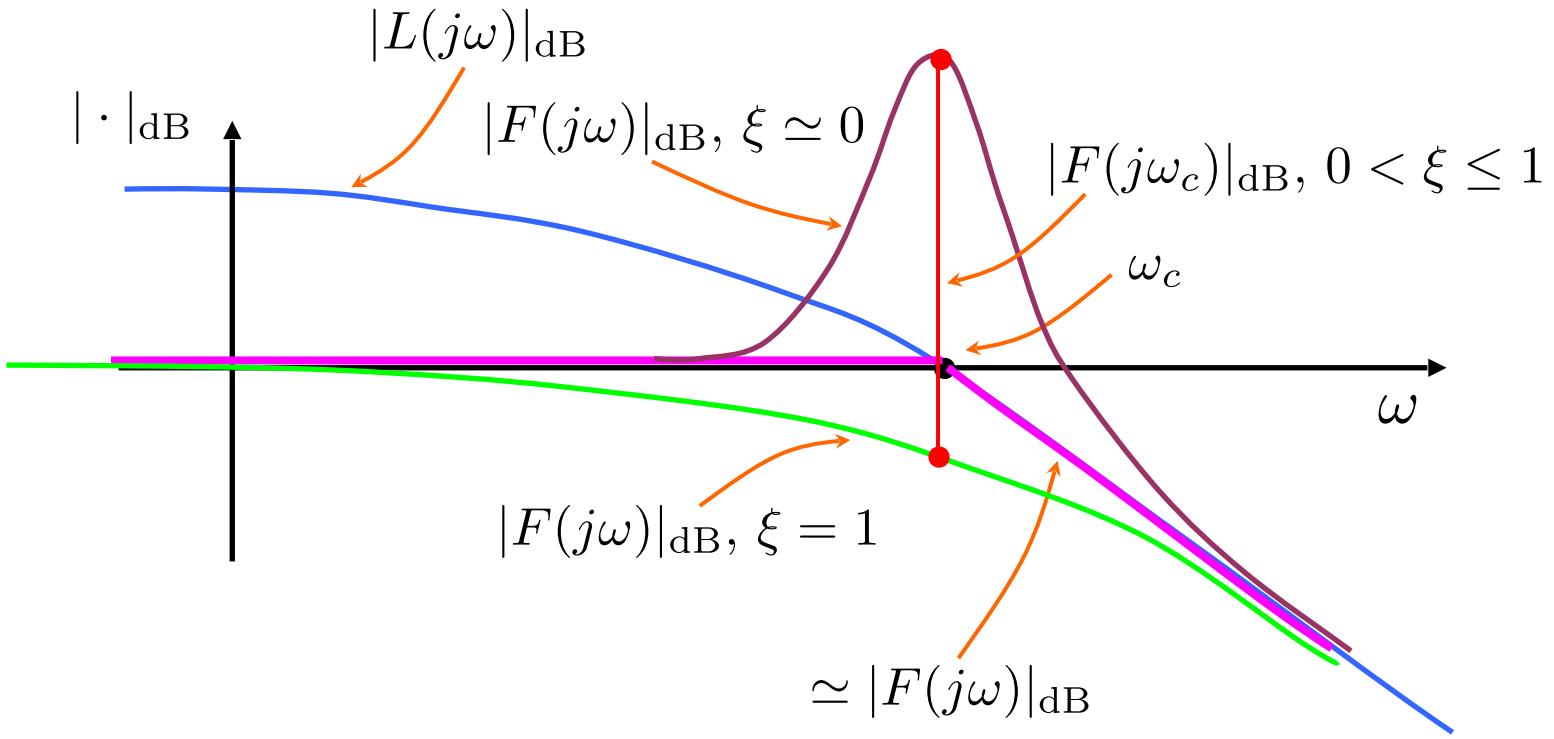


# Closed-Loop Damping Ratio and Phase Margin

But: how about the damping ratio of the dominant poles of  $F(s)$  ?



small damping ratio  $\xi \simeq 0$   $\rightarrow$  "fragile" stability  $\rightarrow$   $\varphi_m \simeq 0$



Question:  $\varphi_m \simeq 0$   $\xrightarrow{?}$   $\xi \simeq 0$

## Closed-Loop Damping Ratio and Phase Margin (contd.)

Let us determine  $|F(j\omega_c)|$  :

$$|L(j\omega_c)| = 1 \rightarrow L(j\omega_c) = 1 \cdot e^{j\varphi_c} \text{ with } \varphi_c = \arg L(j\omega_c)$$

$$\begin{aligned} |F(j\omega_c)| &= \frac{|L(j\omega_c)|}{|1 + L(j\omega_c)|} = \frac{1}{|1 + e^{j\varphi_c}|} = \frac{1}{|1 + \cos \varphi_c + j \sin \varphi_c|} \\ &= \frac{1}{\sqrt{(1 + \cos \varphi_c)^2 + \sin^2 \varphi_c}} = \frac{1}{\sqrt{1 + \cos^2 \varphi_c + 2 \cos \varphi_c + \sin^2 \varphi_c}} \\ &= \frac{1}{\sqrt{2(1 + \cos \varphi_c)}} = \frac{1}{\sqrt{2(1 - \cos \varphi_m)}} \\ &= \frac{1}{2 \sin\left(\frac{\varphi_m}{2}\right)} \quad \text{where } \varphi_m = \varphi_c - (-180^\circ) \end{aligned}$$

Suppose that  $F(s)$  takes on the form of a **second-order** system:

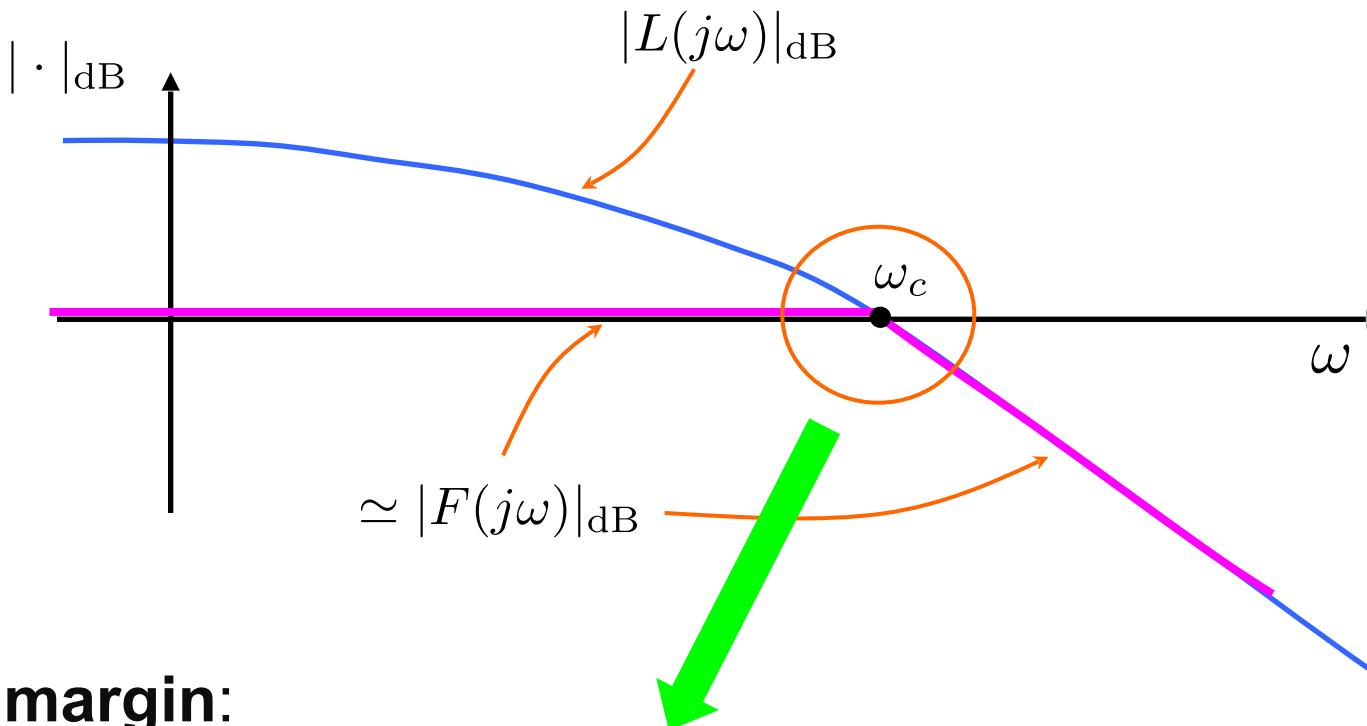
$$F(s) = \frac{\omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2}$$

↳  $|F(j\omega_n)| = \left| \frac{\omega_n^2}{-\omega_n^2 + 2j\xi\omega_n^2 + \omega_n^2} \right| = \left| \frac{1}{j2\xi} \right| = \frac{1}{2\xi}$

Consider the **approximation**  $\omega_n \simeq \omega_c$

↳  $|F(j\omega_c)| = \frac{1}{2 \sin(\frac{\varphi_m}{2})} \simeq |F(j\omega_n)| = \frac{1}{2\xi}$

↳ 
$$\xi \simeq \sin\left(\frac{\varphi_m}{2}\right) \simeq \frac{\varphi_m}{2} \cdot \frac{\pi}{180} \simeq \frac{\varphi_m}{100}$$



**Phase margin:**

- $\varphi_m > 75^\circ$ : one real dominant pole with  $\tau \simeq \frac{1}{\omega_c}$
- $\varphi_m < 75^\circ$ : two complex conjugate poles with  $\omega_n \simeq \omega_c$ ;  $\xi \simeq \frac{\varphi_m}{100}$

# Static Analysis of Sensitivity Function



Using the parametrisation of the open-loop t.f.  $L(s)$  in slides 32-33 Part IV:

$$L(s) = \mu \frac{1}{s^g} \frac{\prod_l \left(1 + \frac{s}{z_l}\right) \prod_h \left(1 + \frac{2\zeta_h}{\alpha_{nh}}s + \frac{1}{\alpha_{nh}^2}s^2\right)}{\prod_i \left(1 + \frac{s}{p_i}\right) \prod_k \left(1 + \frac{2\xi_k}{\omega_{nk}}s + \frac{1}{\omega_{nk}^2}s^2\right)}$$

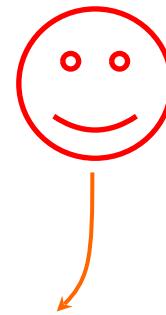
$s \rightarrow 0 \quad 1$

Asymptotic value of the **closed-loop step-response**:

$$\begin{aligned} d(t) &= A \cdot 1(t) \quad \rightarrow \quad y(\infty) &= \lim_{s \rightarrow 0} s \cancel{\cdot} S(s) \cdot \frac{A}{\cancel{s}} = A \lim_{s \rightarrow 0} S(s) \\ &= A \lim_{s \rightarrow 0} \frac{1}{1 + L(s)} = A \lim_{s \rightarrow 0} \frac{1}{1 + \frac{\mu}{s^g}} \\ &= A \lim_{s \rightarrow 0} \frac{s^g}{s^g + \mu} \end{aligned}$$

Therefore:

$$y(\infty) = \begin{cases} A \cdot \frac{1}{1 + \mu} & (\text{hence } \mu_S \simeq 0 \text{ if } \mu \gg 1) \\ 0 & \\ A & (\text{hence } \mu_S = 1) \end{cases} \quad \begin{matrix} \text{if } g = 0 \\ \text{if } g > 0 \\ \text{if } g < 0 \end{matrix}$$



Presence of an integrator (pole = 0) in the direct path of the feedback loop



We have:

$$L(s) = \frac{N(s)}{\varphi(s)} \quad \rightarrow \quad S(s) = \frac{1}{1 + L(s)} = \frac{\varphi(s)}{\varphi(s) + N(s)}$$

Therefore:

- the zeros of  $S(s)$  coincide with the poles of  $L(s)$
- the poles of  $S(s)$  coincide with the roots of  $\varphi(s) + N(s)$

↳ same as for  $F(s)$

# Frequency Response Analysis of Sensitivity Function

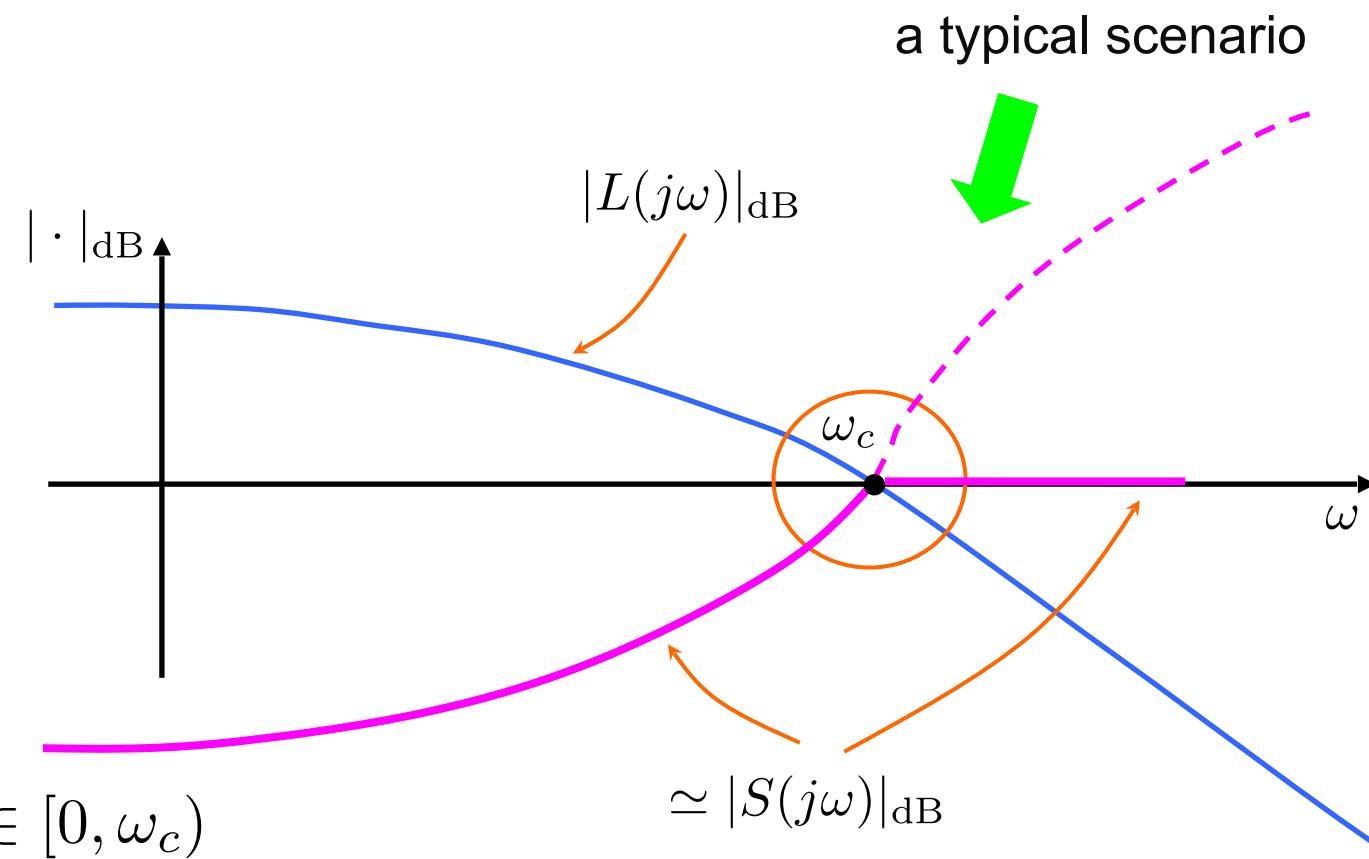
We have:

$$|S(j\omega)| = \frac{1}{|1 + L(j\omega)|} \simeq \begin{cases} 1 & \text{if } |L(j\omega)| \ll 1 \\ \frac{1}{|L(j\omega)|} = -|L(j\omega)|_{\text{dB}} & \text{if } |L(j\omega)| \gg 1 \end{cases}$$

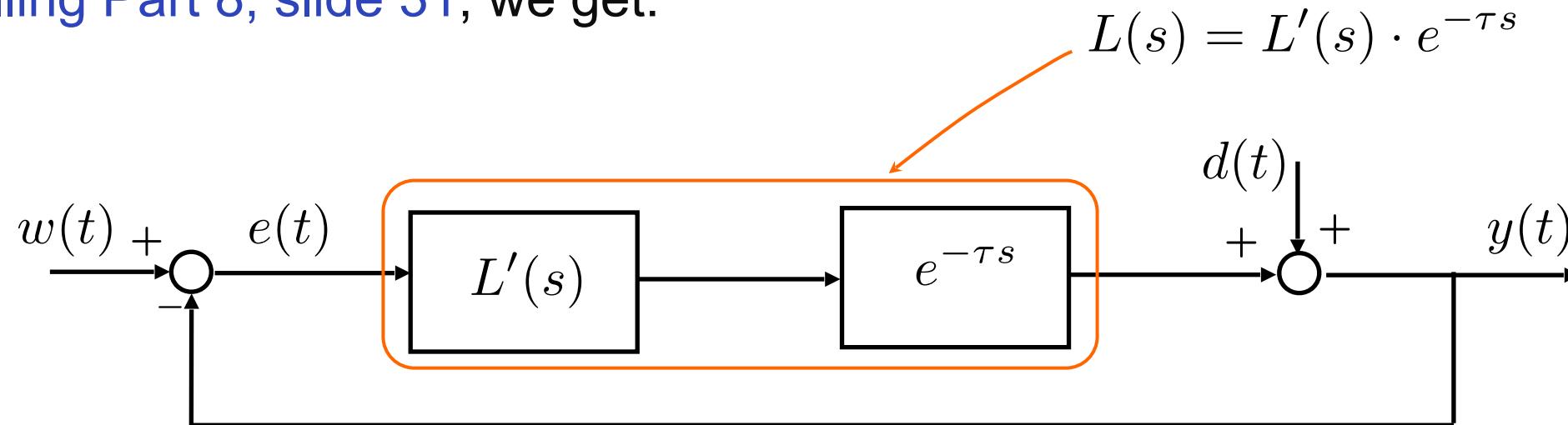
- $S(s)$  **high-pass filter** with bandwidth  $B \simeq [\omega_c, \infty)$
- **Gain:**

$$\mu_S = \begin{cases} 0 & \text{if } g > 0 \\ \frac{1}{1 + \mu} & \text{if } g = 0 \end{cases}$$

- **Disturbance attenuation:**
  - in  $B \simeq [0, \omega_c)$
  - improve attenuation by:
    - increasing  $\omega_c$
    - increasing  $|L(j\omega)|_{\text{dB}}$ ,  $\omega \in [0, \omega_c)$



Recalling Part 8, slide 31, we get:



↳ 
$$\left\{ \begin{array}{l} |L(j\omega)| = |L'(j\omega)| \cdot |e^{j\omega\tau}| = |L'(j\omega)| \\ \arg L(j\omega) = \arg L'(j\omega) - \omega\tau \frac{180}{\pi} \end{array} \right.$$

The presence of a **delay** in a closed-loop system has a significant impact on **stability** and **dynamic performance**.

- The **static analysis is unchanged**:

$$\lim_{s \rightarrow 0} e^{-\tau s} = 1$$

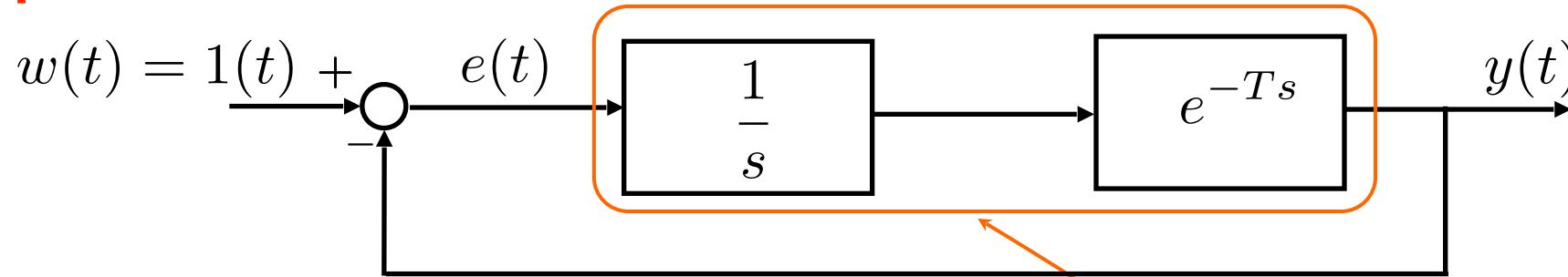
- The delay **modifies the dynamic analysis**:

- the critical angular frequency  $\omega_c$  does not change (the magnitude Bode diagram does not change)
- the critical phase  $\varphi_c$  decreases
- the phase margin  $\varphi_m$  decreases   $\xi$  decreases



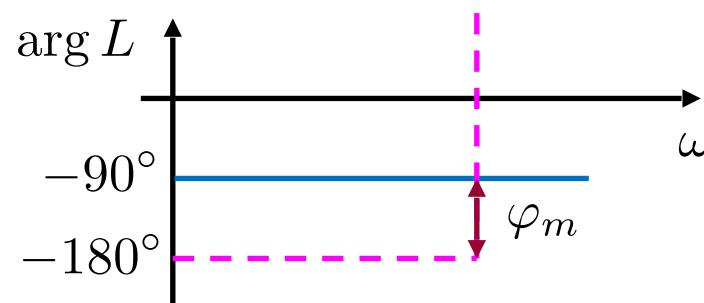
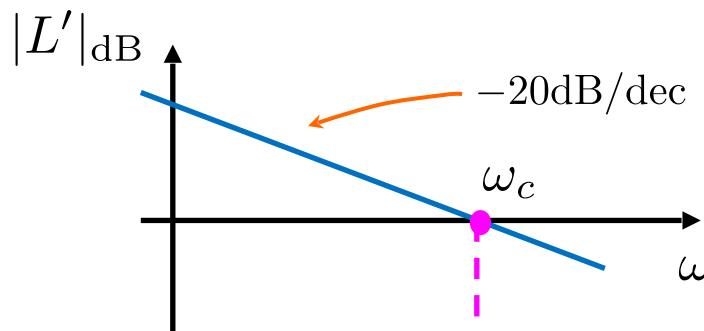
oscillations

## Example



- Case 1:  $T = 0$ ,  $y(\infty) = 1$

$$L(s) = L'(s) \cdot e^{-Ts} = \frac{1}{s} \cdot e^{-Ts}$$



$$\begin{cases} \omega_c = 1 \\ \varphi_m = 90^\circ (\varphi_c = -90^\circ) \end{cases}$$

↙ dominant real pole with  $\tau \simeq \frac{1}{\omega_c} = 1$

## Example (contd.)



- Case 2:  $T > 0$ ,  $y(\infty) = 1$

↳  $\omega_c = 1$

$$\varphi_c = -90^\circ - \omega_c T \cdot \frac{180^\circ}{\pi}$$

$$= -90^\circ - T \cdot \frac{180^\circ}{\pi}$$

↳  $\varphi_m = 90^\circ - T \cdot \frac{180^\circ}{\pi}$

Hence:

closed-loop asymptotic stability



$$T < \frac{\pi}{2} \simeq 1.57 \text{ sec}$$

Consider the specific case:  $T = 1$

↳  $\varphi_m = 90^\circ - T \cdot \frac{180^\circ}{\pi} = 90^\circ - 57^\circ = 33^\circ$

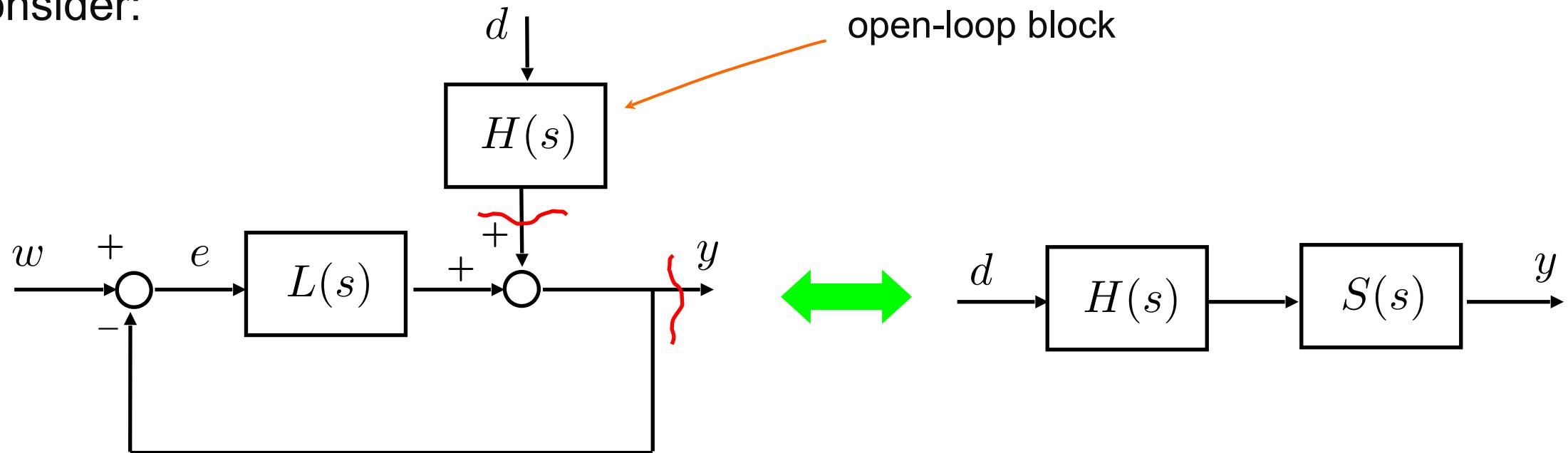
Hence, recalling slide 43:

- $\varphi_m > 75^\circ$ : one dominant real pole with  $\tau \simeq \frac{1}{\omega_c}$
- $\varphi_m < 75^\circ$ : two dominant complex conjugate poles with  $\omega_n \simeq \omega_c$ ;  $\xi \simeq \frac{\varphi_m}{100}$

↳  $\left\{ \begin{array}{l} \omega_n \simeq \omega_c = 1 \\ \xi \simeq \frac{\varphi_m}{100} \simeq 0.33 \end{array} \right.$  →  $t_s \simeq \frac{4.6}{\xi \omega_n} \simeq 14 \text{ sec !!!}$

# Presence of Open-Loop Blocks

Consider:



Clearly:

closed-loop asymptotic stability

- $\iff \begin{cases} H(s) \text{ asymptotically stable} \\ \quad \Leftrightarrow \text{Re (poles)} < 0 \\ S(s) \text{ asymptotically stable} \\ \quad \Leftrightarrow \text{Bode, Nyquist} \end{cases}$

# Analysis of the Effect of the Disturbance on the Output



Clearly:  $R(s) = \frac{H(s)}{1 + L(s)} = H(s) \cdot S(s)$

- **Static Analysis:**

Without loss of generality, suppose  $g_H = 0$ ,  $g_L = 0$  :

$$\begin{aligned} d(t) = A \cdot 1(t) &\xrightarrow{\text{ }} y(\infty) = \lim_{s \rightarrow 0} s \cdot R(s) \cdot \frac{A}{s} = A \lim_{s \rightarrow 0} R(s) \\ &= A \cdot \frac{\mu_H}{1 + \mu_L} = A \cdot \mu_H \cdot \mu_S \end{aligned}$$

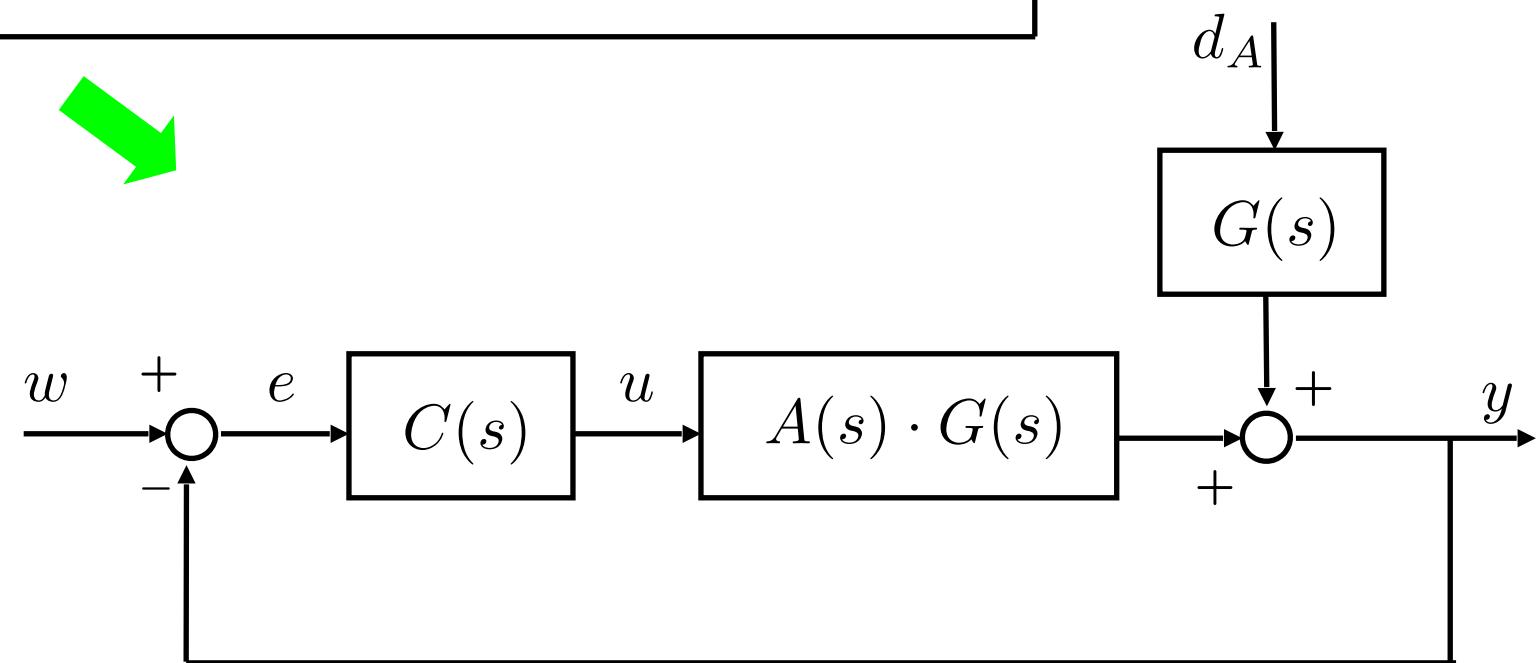
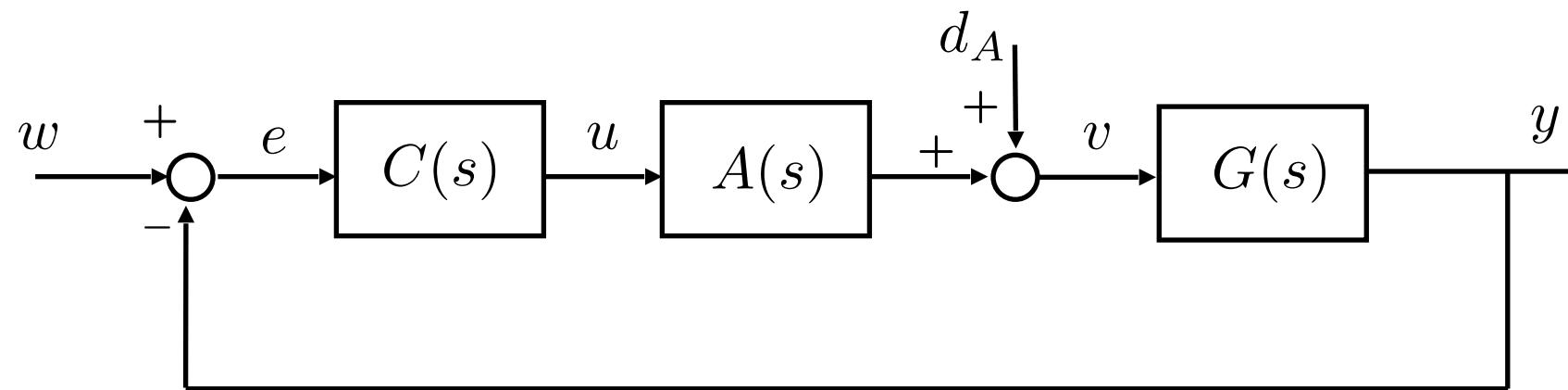
- **Dynamic Analysis:**

$$|R(j\omega)| = |H(j\omega)| \cdot |S(j\omega)| \xrightarrow{\text{ }} |R(j\omega)|_{\text{dB}} = |H(j\omega)|_{\text{dB}} + |S(j\omega)|_{\text{dB}}$$

# Disturbance on the Actuators



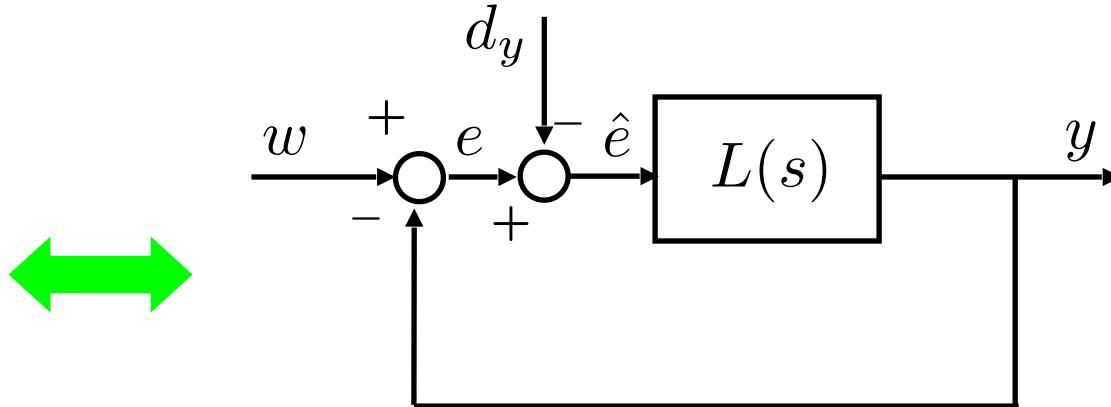
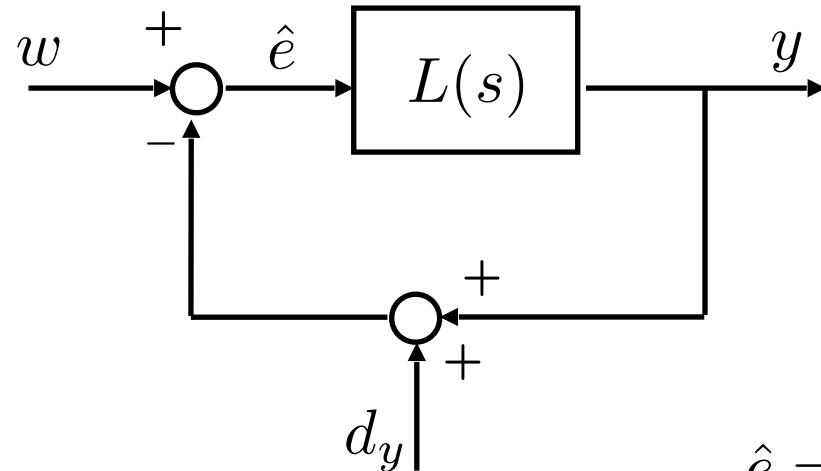
Recalling slide 10 (suppose  $M(s) = 1$  for simplicity):



# Disturbance on the Output Measurements



This scenario represent the presence of disturbances on sensors, very frequent in applications



Hence:

$$\hat{e} = w - (y + d_y) = w - y - d_y = e - d_y$$

$$\frac{E(s)}{D_y(s)} = \frac{L(s)}{1 + L(s)} = F(s)$$

**low-pass filter with bandwidth  $B \simeq [0, \omega_c]$**



**low-frequency** disturbances  $d_y(t)$   
influence **directly** the error  $e(t)$

- **Static Analysis:**

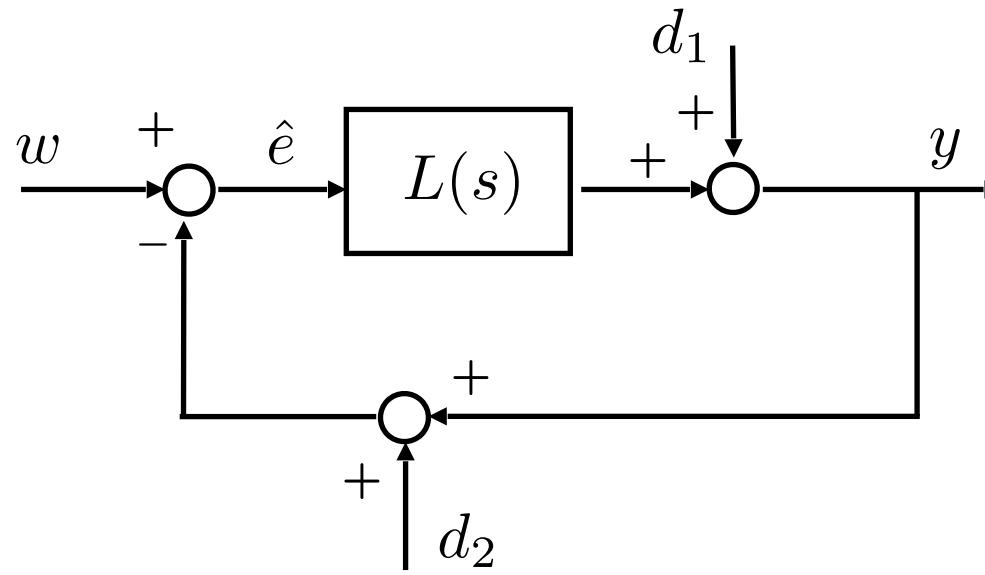
$$d_y(t) = A \cdot 1(t)$$

↳  $e(\infty) = \lim_{s \rightarrow 0} s \cdot F(s) \frac{1}{s} = \begin{cases} A & \text{if } g > 0 \\ A \cdot \frac{\mu}{1 + \mu} & \text{if } g = 0 \end{cases}$

Hence, rejection of disturbances on the direct path is in contrast with rejection of disturbances on the feedback path



Consider:



A standard feedback control system allows to reject simultaneously:

- **low-frequency** disturbances  $d_1$  on the **direct** path with  $\omega \ll \omega_c$
- **high-frequency** disturbances  $d_2$  on the **feedback** path with  $\omega \gg \omega_c$

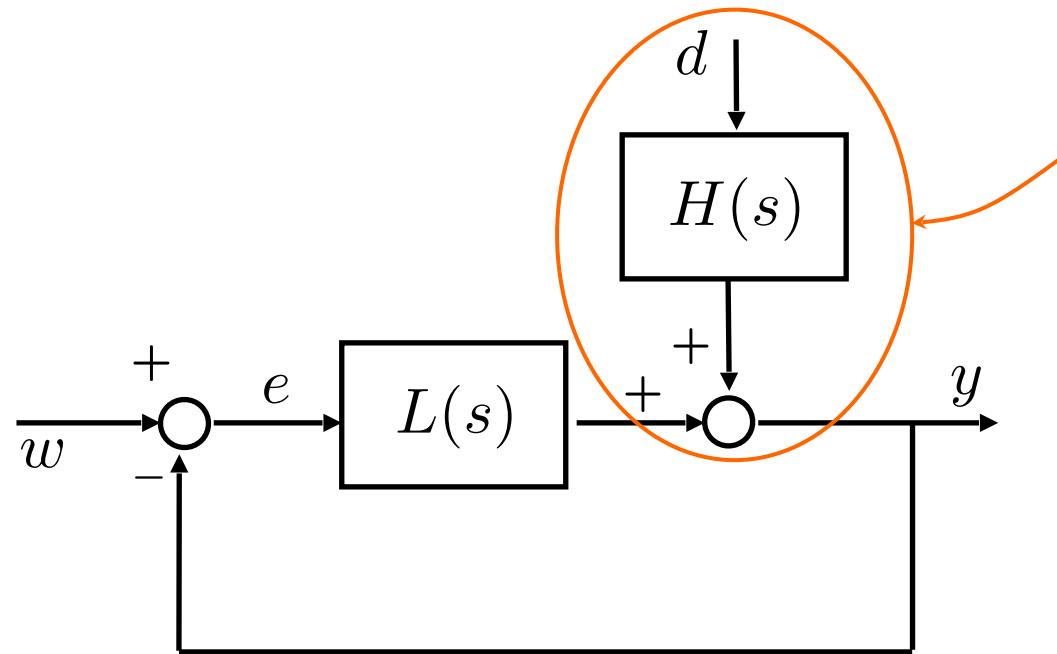
# **Analysis of Feedback Control Systems**

## **Methods and Tools in the s-Domain:**

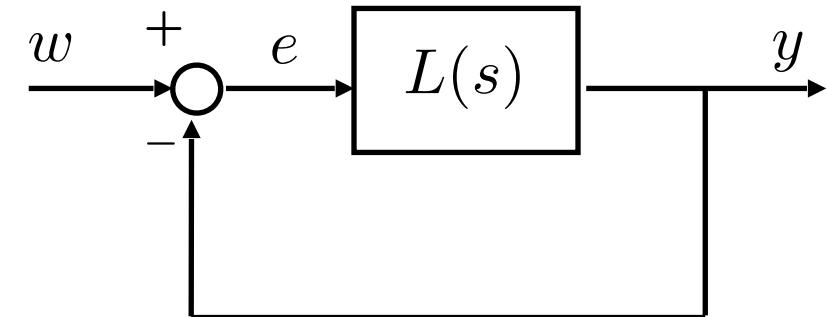
### **The Root-Locus Technique**

# The Root Locus Technique - Basics and Definitions

Refer to the usual scheme feedback control scheme:



Open-loop block not affecting closed-loop stability analysis



The root locus technique is a *graphical method* relating the **location in the complex plane of the closed-loop poles**, that is, **the roots of**

$$1 + L(s) = 0$$

with the location in the complex plane of the open-loop zeros and poles and the **open-loop gain**.

The RL technique has distinct advantages and disadvantages in analysing and designing feedback control systems with respect to frequency domain tools:

- RL enables to obtain a graphical evidence on the location of **all** closed-loop poles and not only the location of the closed-loop **dominant** poles as is for frequency domain techniques (see slides [Part 9, 40-44](#))
- RL technique also can be applied in the presence of **unstable open-loop poles**
- More in general, the RL tool can be applied in many scenarios in which the Bode criterion is not applicable
- The RL cannot be applied when the open-loop transfer function  $L(s)$  is not rational (i.e., a fraction of polynomial numerator and denominator). The typical case is when  $L(s)$  contains **delay blocks**

# The Root Locus Technique - Details and Examples



We refer to the following parameterisation of  $L(s)$  using poles and zeros (see slide Part 4, 28):

$$L(s) = \varrho \frac{(s + z_1)(s + z_2) \cdots (s + z_m)}{(s + p_1)(s + p_2) \cdots (s + p_n)} = \varrho \frac{\prod_{j=1}^m (s + z_j)}{\prod_{i=1}^n (s + p_i)}$$

The RL diagram shows in the complex plane for all  $\varrho \in \mathbb{R}$ ,  $\varrho \neq 0$  the roots of

$$1 + L(s) = 1 + \varrho \frac{(s + z_1)(s + z_2) \cdots (s + z_m)}{(s + p_1)(s + p_2) \cdots (s + p_n)} = 1 + \varrho \frac{\prod_{j=1}^m (s + z_j)}{\prod_{i=1}^n (s + p_i)} = 0$$

- The **direct RL** refers to all values  $\varrho > 0$
- The **inverse RL** refers to all values  $\varrho < 0$
- For  $\varrho = 0$  the feedback loop ``degenerates'' and the closed-loop poles coincide with the open-loop ones



## Illustrative Example:

Consider the open-loop transfer function  $L(s) = \varrho \frac{s+2}{s(s+1)}$ . We have:

$$1 + L(s) = 1 + \varrho \frac{s+2}{s(s+1)} = \frac{s(s+1) + \varrho(s+2)}{s(s+1)} = \frac{s^2 + (1+\varrho)s + 2\varrho}{s(s+1)}$$

Hence, the closed-loop poles coincide with the roots of the equation

$$s^2 + (1 + \varrho)s + 2\varrho = 0$$

and the location in the complex plane of these roots changes when  $\varrho$  changes thus giving rise to **changes in closed-loop stability** properties, as well as **changes in characteristics of the closed-loop transient behaviours**.

# The Root Locus Technique - Mathematical Characterisation



We have:

$$1 + L(s) = 0 \quad \rightarrow \quad 1 + \varrho \frac{\prod_{j=1}^m (s + z_j)}{\prod_{i=1}^n (s + p_i)} = 0 \quad \rightarrow \quad \frac{\prod_{j=1}^m (s + z_j)}{\prod_{i=1}^n (s + p_i)} = -\frac{1}{\varrho}$$

Thus:

- $\frac{\prod_{j=1}^m |s + z_j|}{\prod_{i=1}^n |s + p_i|} = \frac{1}{|\varrho|}$
- $\arg \prod_{j=1}^m (s + z_j) - \arg \prod_{i=1}^n (s + p_i) = \begin{cases} (2k+1) \cdot 180^\circ, & k \in \mathbb{Z}, \\ 2k \cdot 180^\circ, & k \in \mathbb{Z}, \end{cases} \quad \begin{array}{l} \text{if } \varrho > 0 \text{ (Direct RL)} \\ \text{if } \varrho < 0 \text{ (Inverse RL)} \end{array}$

- Relationship  $\frac{\prod_{j=1}^m |s + z_j|}{\prod_{i=1}^n |s + p_i|} = \frac{1}{|\varrho|}$  associates every RL point with the values of  $\varrho$
- Relationships

$$\arg \prod_{j=1}^m (s + z_j) - \arg \prod_{i=1}^n (s + p_i) = \begin{cases} (2k + 1) \cdot 180^\circ, k \in \mathbb{Z}, & \text{if } \varrho > 0 \text{ (Direct RL)} \\ 2k \cdot 180^\circ, k \in \mathbb{Z}, & \text{if } \varrho < 0 \text{ (Inverse RL)} \end{cases}$$

fully characterise the shape of the Direct and Inverse RLs in the complex plane

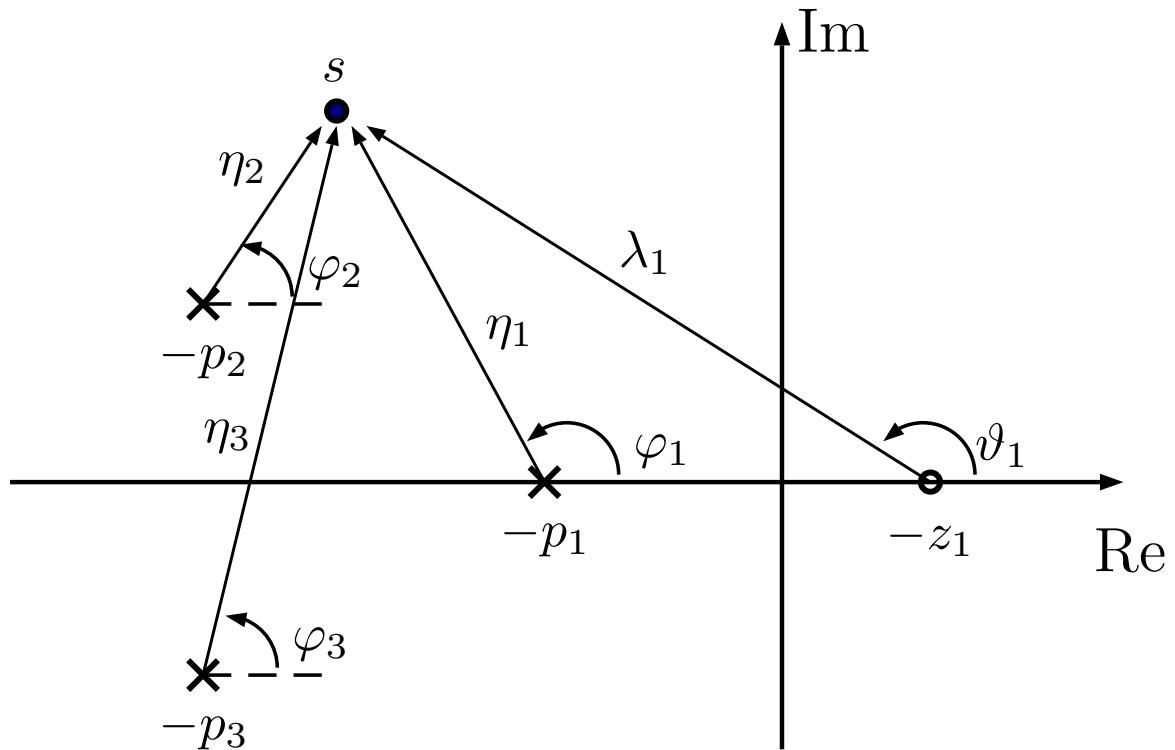
# The Root Locus Technique - Geometric Interpretation

From

$$\frac{\prod_{j=1}^m |s + z_j|}{\prod_{i=1}^n |s + p_i|} = \frac{1}{|\varrho|}$$

we get

$$\frac{\prod_{j=1}^m |s + z_j|}{\prod_{i=1}^n |s + p_i|} = \frac{\prod_{j=1}^m \lambda_j}{\prod_{i=1}^n \eta_i} = \frac{1}{|\varrho|}$$



Moreover, from

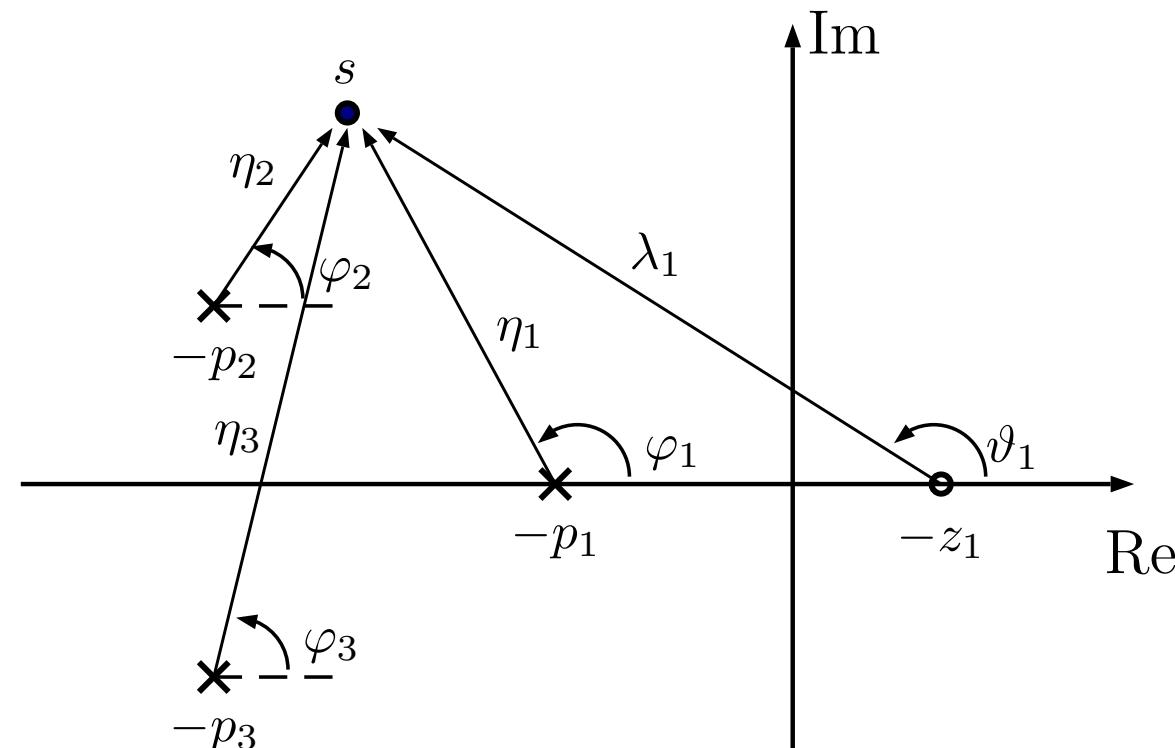
$$\arg \prod_{j=1}^m (s + z_j) = \sum_{j=1}^m \arg (s + z_j) = \sum_{j=1}^m \theta_j$$

and

$$\arg \prod_{i=1}^n (s + p_i) = \sum_{i=1}^n \arg (s + p_i) = \sum_{j=1}^n \varphi_i$$

we get

$$\sum_{j=1}^m \theta_j - \sum_{i=1}^n \varphi_i = \begin{cases} (2k+1) \cdot 180^\circ, k \in \mathbb{Z}, \\ 2k \cdot 180^\circ, k \in \mathbb{Z}, \end{cases}$$



if  $\varrho > 0$  (**Direct RL**)  
if  $\varrho < 0$  (**Inverse RL**)

## Illustrative Example:

Consider the open-loop transfer function:

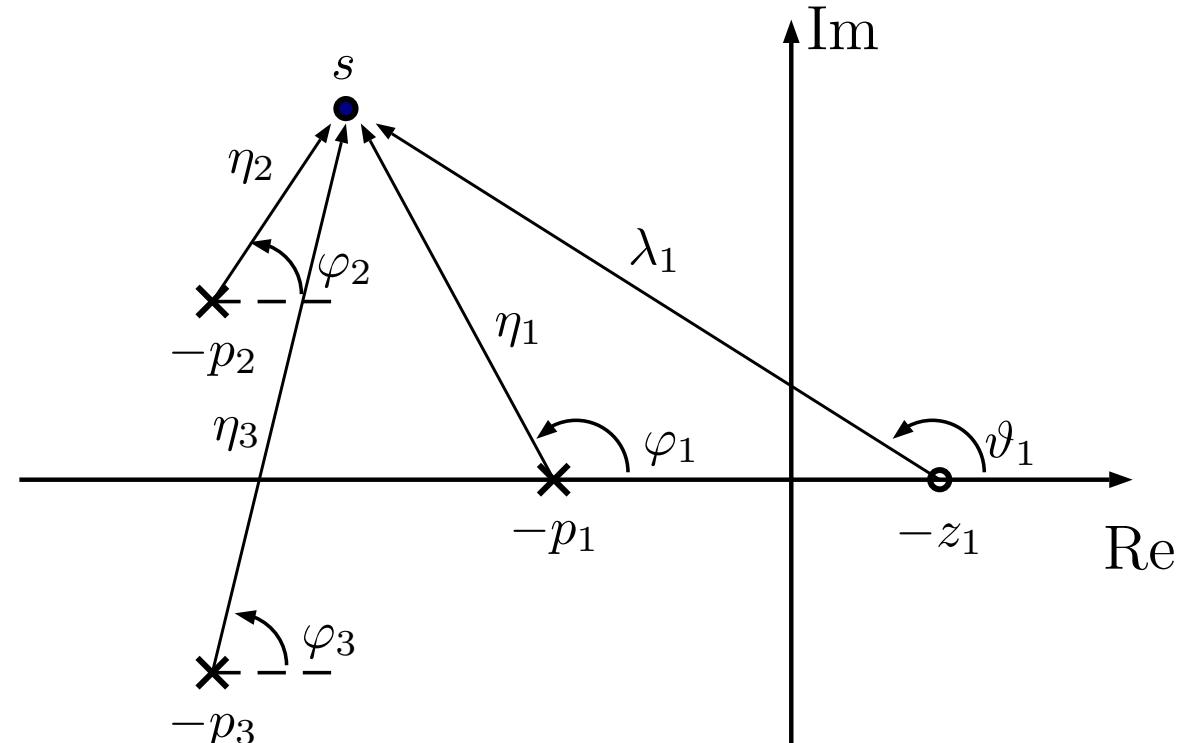
$$L(s) = \varrho \frac{s - 1}{(s + 2)(s^2 + 6s + 13)}$$

The value of  $\varrho$  for the specific point  $s$  in the complex plane is:

$$|\varrho| = \frac{1}{\lambda_1} \prod_{i=1}^3 \eta_i$$

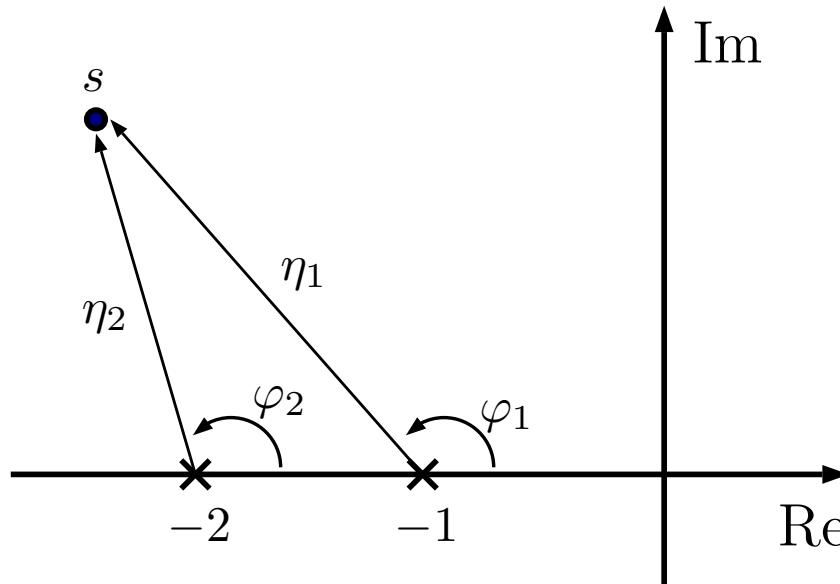
and

- the specific point  $s$  in the complex plane belongs to the Direct RL if  $\theta_1 - \sum_{i=1}^3 \varphi_i = (2k + 1) \cdot 180^\circ, k \in \mathbb{Z}$
- the specific point  $s$  in the complex plane belongs to the Inverse RL if  $\theta_1 - \sum_{i=1}^3 \varphi_i = 2k \cdot 180^\circ, k \in \mathbb{Z}$



## A Simple Example

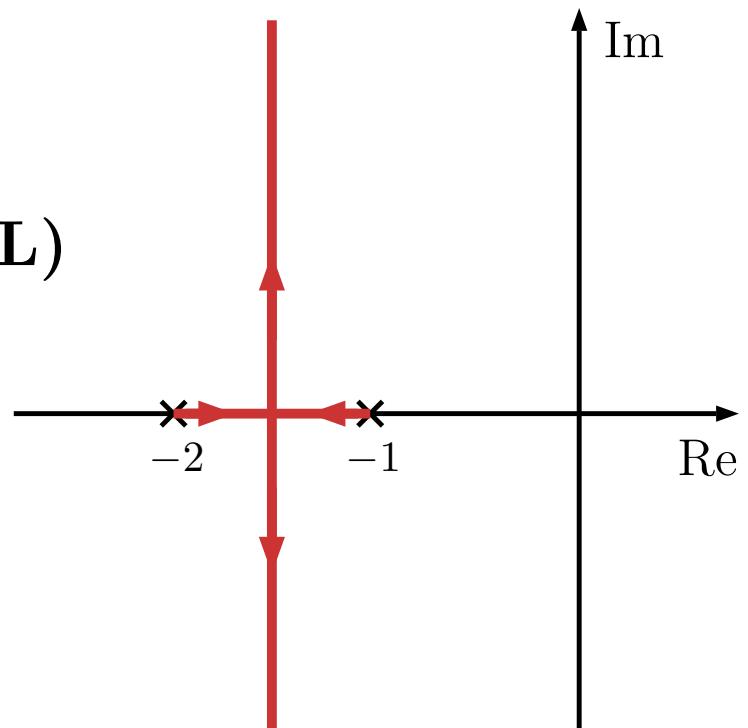
$$L(s) = \varrho \frac{1}{(s+1)(s+2)}$$



We have:

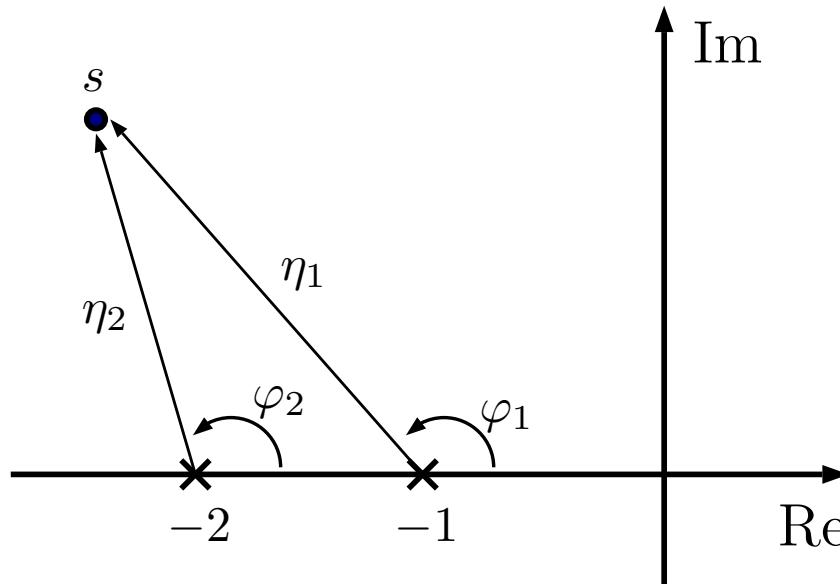
$$-\varphi_1 - \varphi_2 = (2k + 1) \cdot 180^\circ, k \in \mathbb{Z}, \text{ if } \varrho > 0 \text{ (Direct RL)}$$

The arrows on the Direct RL branches depicted in **red** show the directions of movement of the closed-loop poles for increasing values of  $\varrho > 0$



## A Simple Example (contd.)

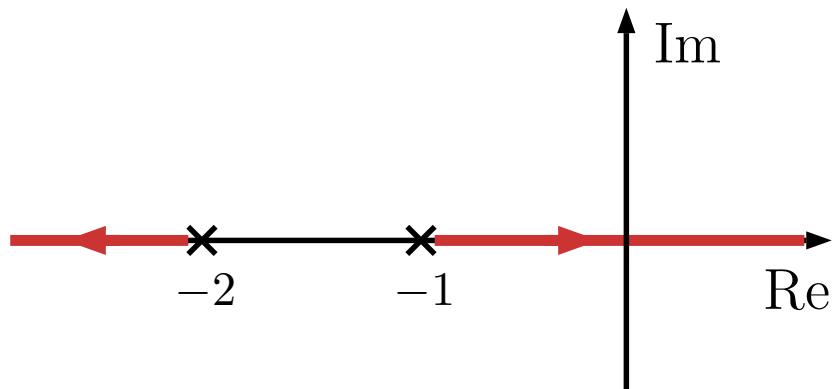
$$L(s) = \varrho \frac{1}{(s+1)(s+2)}$$



We have:

$$-\varphi_1 - \varphi_2 = 2k \cdot 180^\circ, k \in \mathbb{Z}, \text{ if } \varrho < 0 \text{ (Inverse RL)}$$

The arrows on the Inverse RL branches depicted in **red** show the directions of movement of the closed-loop poles for decreasing values of  $\varrho < 0$  (that is, increasing values of  $|\varrho|$  ).





Starting from  $L(s) = \varrho \frac{1}{(s+1)(s+2)}$  we have

$$1 + L(s) = 1 + \varrho \frac{1}{(s+1)(s+2)} = \frac{(s+1)(s+2) + \varrho}{(s+1)(s+2)} = \frac{s^2 + 3s + (2 + \varrho)}{(s+1)(s+2)}$$

that gives the solutions  $s_{1,2} = -\frac{3}{2} \pm \sqrt{1 - 4\varrho}$

- However, this is **not** the procedure that is typically carried out to draw the RL.
- Instead, the construction of the RL typically follows some **basic properties** shown in the subsequent slides.

# The Root Locus Technique - Basic Properties



Consider again:

$$1 + L(s) = 0 \quad \rightarrow \quad 1 + \varrho \frac{\prod_{j=1}^m (s + z_j)}{\prod_{i=1}^n (s + p_i)} = 0 \quad \rightarrow \quad \prod_{i=1}^n (s + p_i) + \varrho \prod_{j=1}^m (s + z_j) = 0$$

- **Property 1:** the RL is made of  $2n$  branches:  $n$  branches constitute the **Direct RL** and  $n$  branches constitute the **Inverse RL**
- **Property 2:** the RL is symmetric with respect to the real axis
- **Property 3:** when  $|\varrho| \rightarrow 0$  the RL branches "originate" from the open-loop poles, that is the poles of  $L(s)$
- **Property 4:** when  $|\varrho| \rightarrow \infty$ ,  $m$  branches of the RL "terminate" on the open-loop zeros, that is the zeros of  $L(s)$  and  $n - m$  branches diverge to infinity



- **Property 5:** the  $n - m$  branches diverging to infinity asymptotically approach asymptotes having the following characteristics:
  - the  $n - m$  asymptotes intersect the real axis on the point  $x_a$  (named **centroid**) as follows:
$$x_a = \frac{1}{n-m} \left( \sum_{j=1}^m z_j - \sum_{i=1}^n p_i \right)$$
  - the  $n - m$  asymptotes form the following angles with the real axis:

$$\psi_a = \begin{cases} \frac{(2k+1) \cdot 180^\circ}{n-m}, & k = 0, \dots, n-m-1, \quad \text{if } \varrho > 0 \text{ (Direct RL)} \\ \frac{2k \cdot 180^\circ}{n-m}, & k = 0, \dots, n-m-1, \quad \text{if } \varrho < 0 \text{ (Inverse RL)} \end{cases}$$



- **Property 6:** all points on the real axis belong to the RL (Direct and Inverse), excluding the open-loop zeros and poles of  $L(s)$
- **Property 7:** all points on the real axis having on their right-hand-side an odd number of open-loop zeros and poles of  $L(s)$  belong to the **Direct RL**
- **Property 8:** all points on the real axis having on their right-hand-side an even number of open-loop zeros and poles of  $L(s)$  belong to the **Inverse RL**
- **Property 9:** possible intersections  $\bar{x} \in \mathbb{R}$  of the RL with the real axis can be obtained as follows:

$$\bar{x} \in \mathbb{R} \text{ such that } \left. \frac{d\gamma(x)}{dx} \right|_{x=\bar{x}} = 0 \quad \text{where} \quad \gamma(x) = -\frac{\prod_{i=1}^n (x + p_i)}{\prod_{j=1}^m (x + z_j)}$$

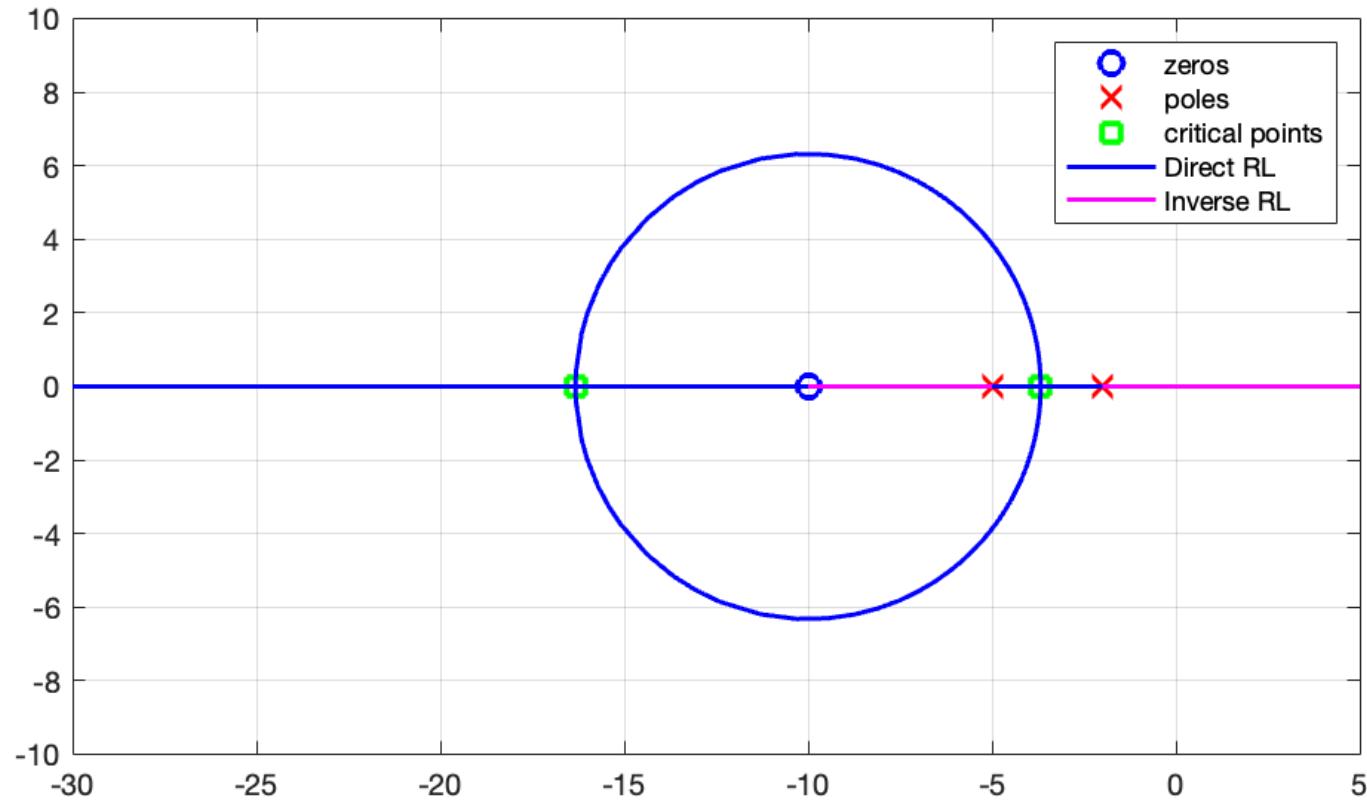
that is,  $\bar{x} \in \mathbb{R}$  are **stationary points** (if any) of  $\gamma(x)$

## Example 1

$$L(s) = \frac{s + 10}{(s + 5)(s + 2)}$$

### Remarks:

- The open-loop zero and poles are all located in the left half-plane. Hence, as shown by the **Direct RL**, the closed-loop systems is asymptotically stable for any  $\varrho > 0$
- As shown by the **Inverse RL**, decreasing  $\varrho < 0$  makes the closed-loop system unstable
- In the **Direct RL**, the zero "attracts" the RL for increasing  $\varrho > 0$
- In the **Direct RL**, increasing  $\varrho > 0$  causes first a decrease, then an increase of the damping ratio, eventually yielding real closed-loop poles for high values of  $\varrho > 0$



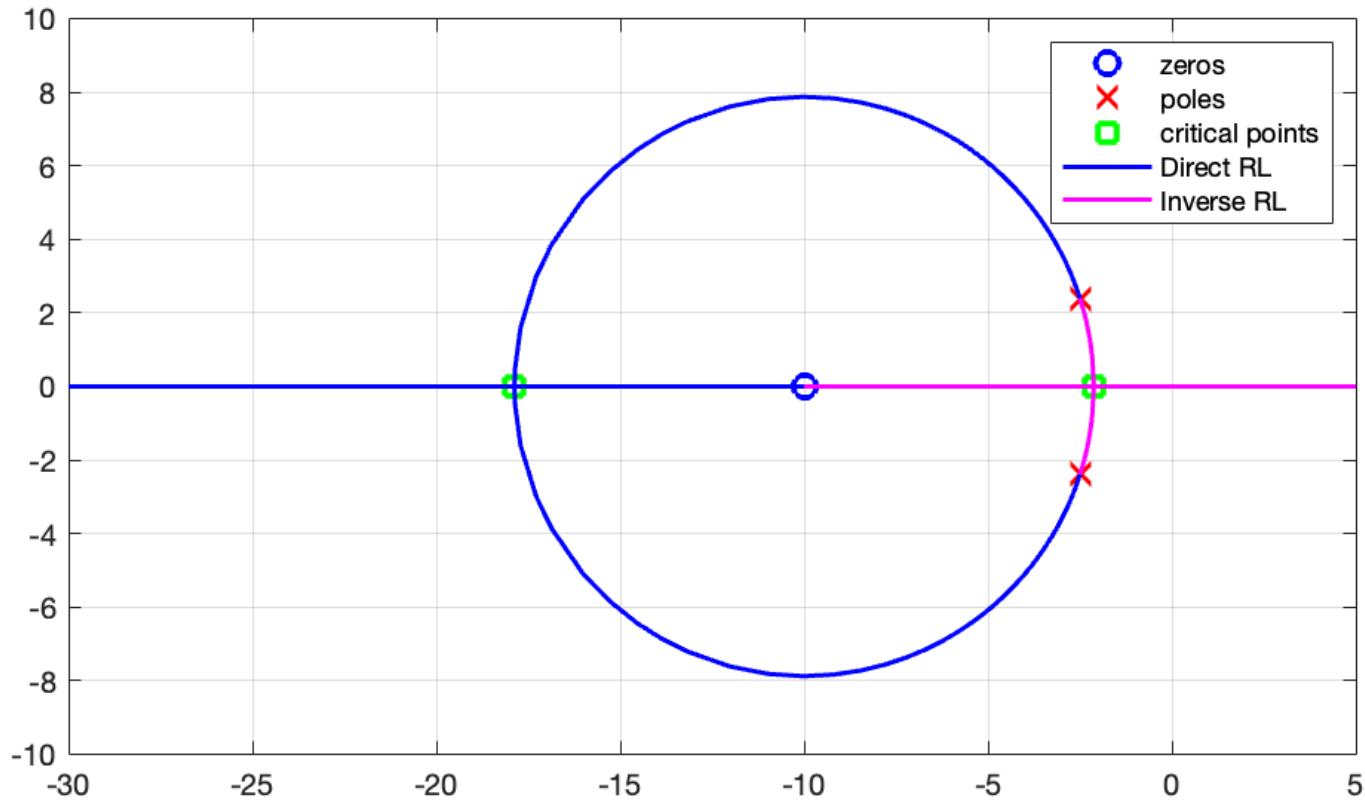
- Both critical points belong to the **Direct RL**

## Example 2

$$L(s) = \frac{s + 10}{s^2 + 5s + 12}$$

### Remarks:

- The open-loop zero and poles are all located in the left half-plane. Hence, as shown by the **Direct RL**, the closed-loop systems is asymptotically stable for any  $\varrho > 0$
- As shown by the **Inverse RL**, decreasing  $\varrho < 0$  makes the closed-loop system unstable
- In the **Direct RL**, the zero "attracts" the RL for increasing  $\varrho > 0$
- In the **Direct RL**, increasing  $\varrho > 0$  causes first a decrease, then an increase of the damping ratio, eventually yielding real closed-loop poles for high values of  $\varrho > 0$



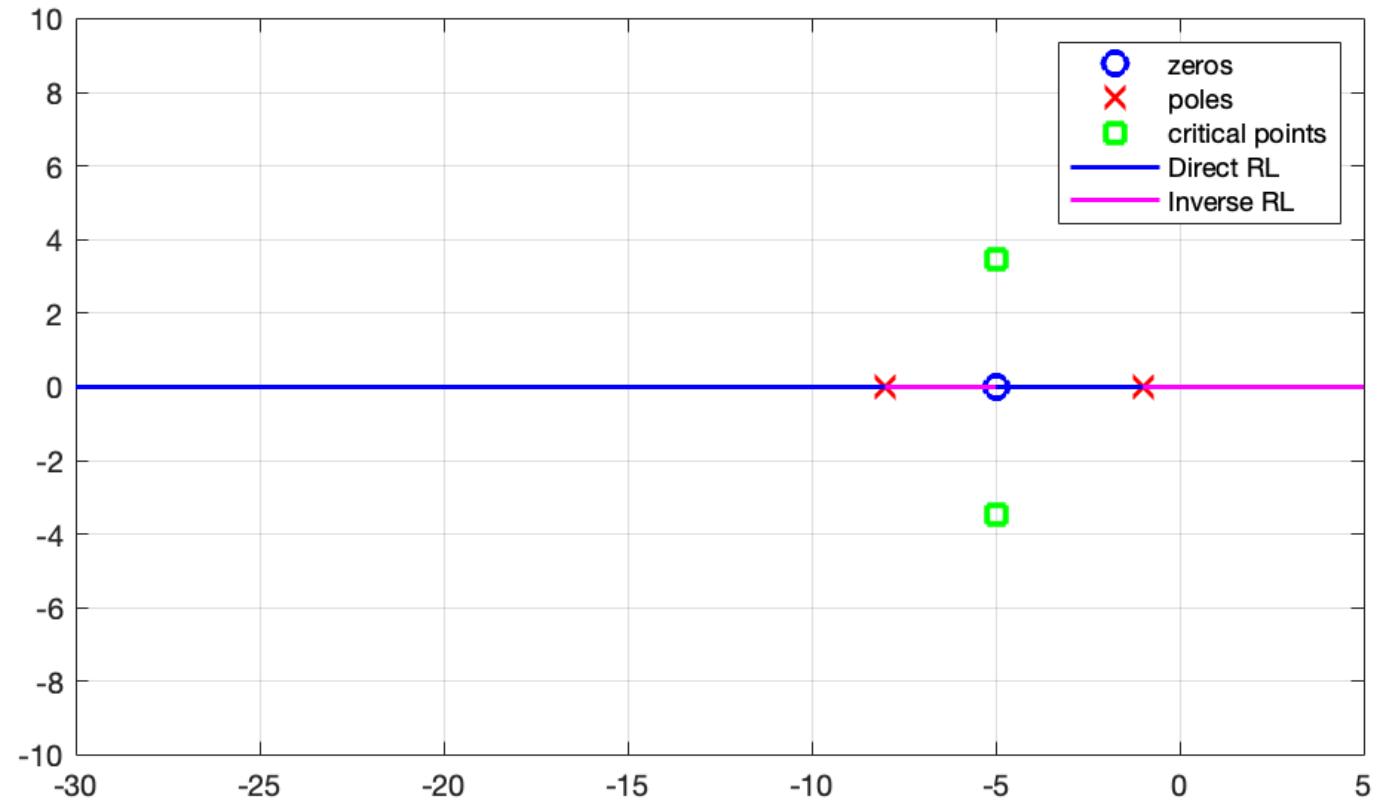
- One critical point belongs to the **Direct RL** and one to the **Inverse RL**

## Example 3

$$L(s) = \frac{s + 5}{(s + 1)(s + 8)}$$

### Remarks:

- The open-loop zero and poles are all located in the left half-plane. Hence, as shown by the **Direct RL**, the closed-loop systems is asymptotically stable for any  $\varrho > 0$
- As shown by the **Inverse RL**, decreasing  $\varrho < 0$  makes the closed-loop system unstable
- In the **Direct RL**, the zero "attracts" the RL for increasing  $\varrho > 0$
- Direct and Inverse RLs** develop only on the real axis
- Critical points do **not** belong to the RL (neither **Direct** nor **Inverse**)

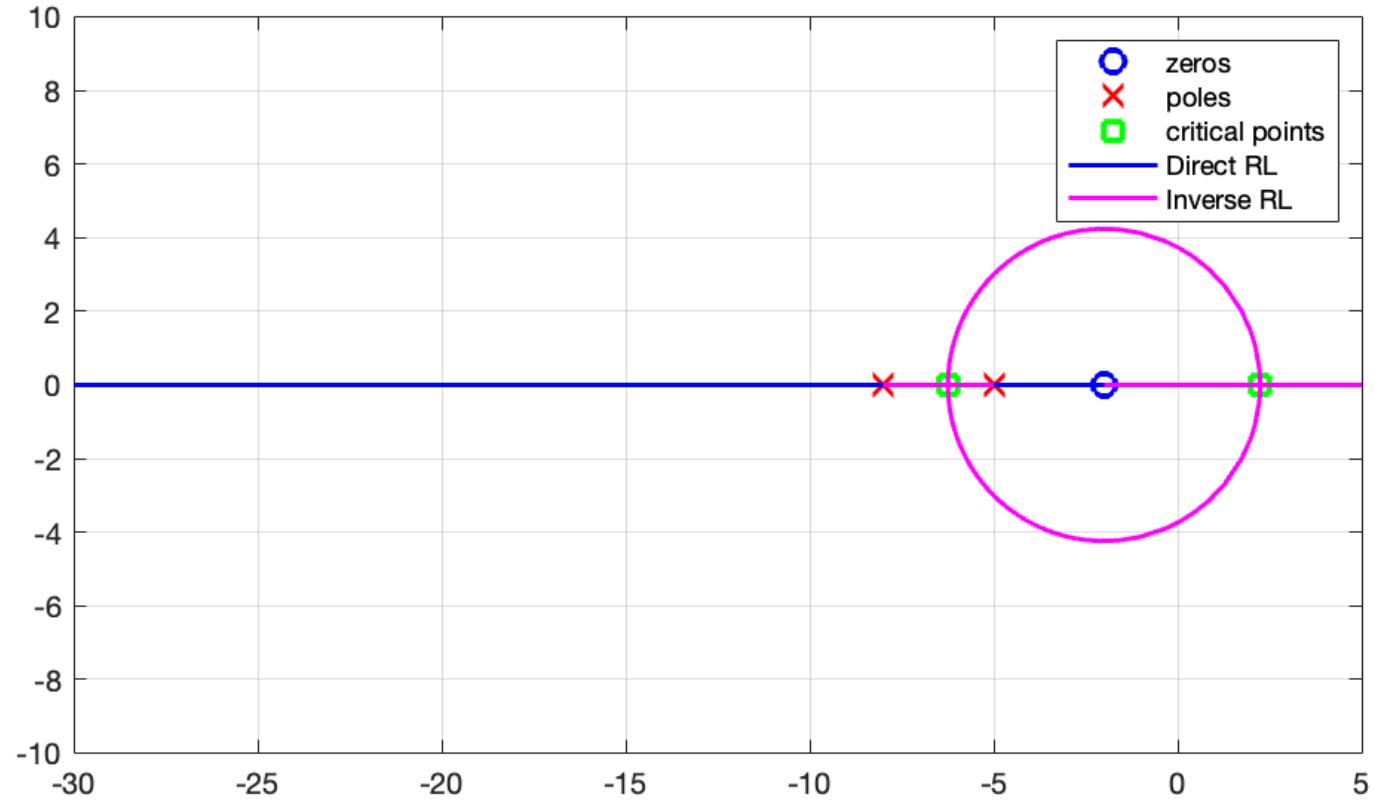


## Example 4

$$L(s) = \frac{s + 2}{(s + 5)(s + 8)}$$

### Remarks:

- The open-loop zero and poles are all located in the left half-plane. Hence, as shown by the **Direct RL**, the closed-loop systems is asymptotically stable for any  $\varrho > 0$
- As shown by the **Inverse RL**, decreasing  $\varrho < 0$  makes the closed-loop system unstable
- In the **Direct RL**, the zero "attracts" the RL for increasing  $\varrho > 0$
- The **Direct RL**, fully belongs to the real axis, hence closed-loop poles are all real and negative for any value of  $\varrho > 0$



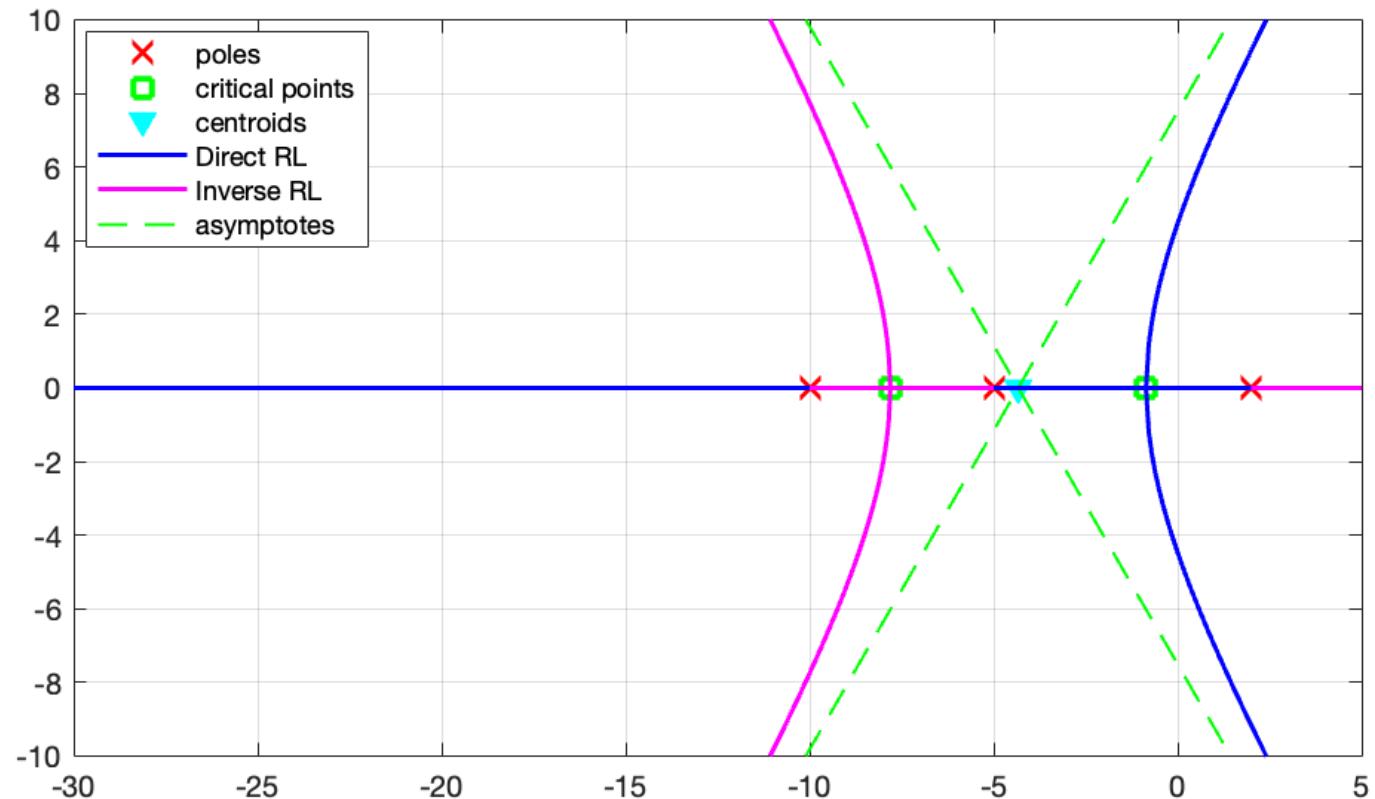
- Unlike Example 1, both critical points now belong to the **Inverse RL**

## Example 5

$$L(s) = \frac{1}{(s - 2)(s + 5)(s + 10)}$$

### Remarks:

- There are no open-loop zeros and one of the open-loop poles is located in the right half-plane.
- Both the **Direct RL** and the **Inverse RL** have three asymptotes forming angles of  $120^\circ$  between them
- As shown by the **Direct RL**, increasing  $\varrho > 0$  makes the closed-loop system unstable
- One branch of the **Inverse RL**, fully belongs to the positive real axis, hence for any  $\varrho < 0$ , the closed-loop system is unstable



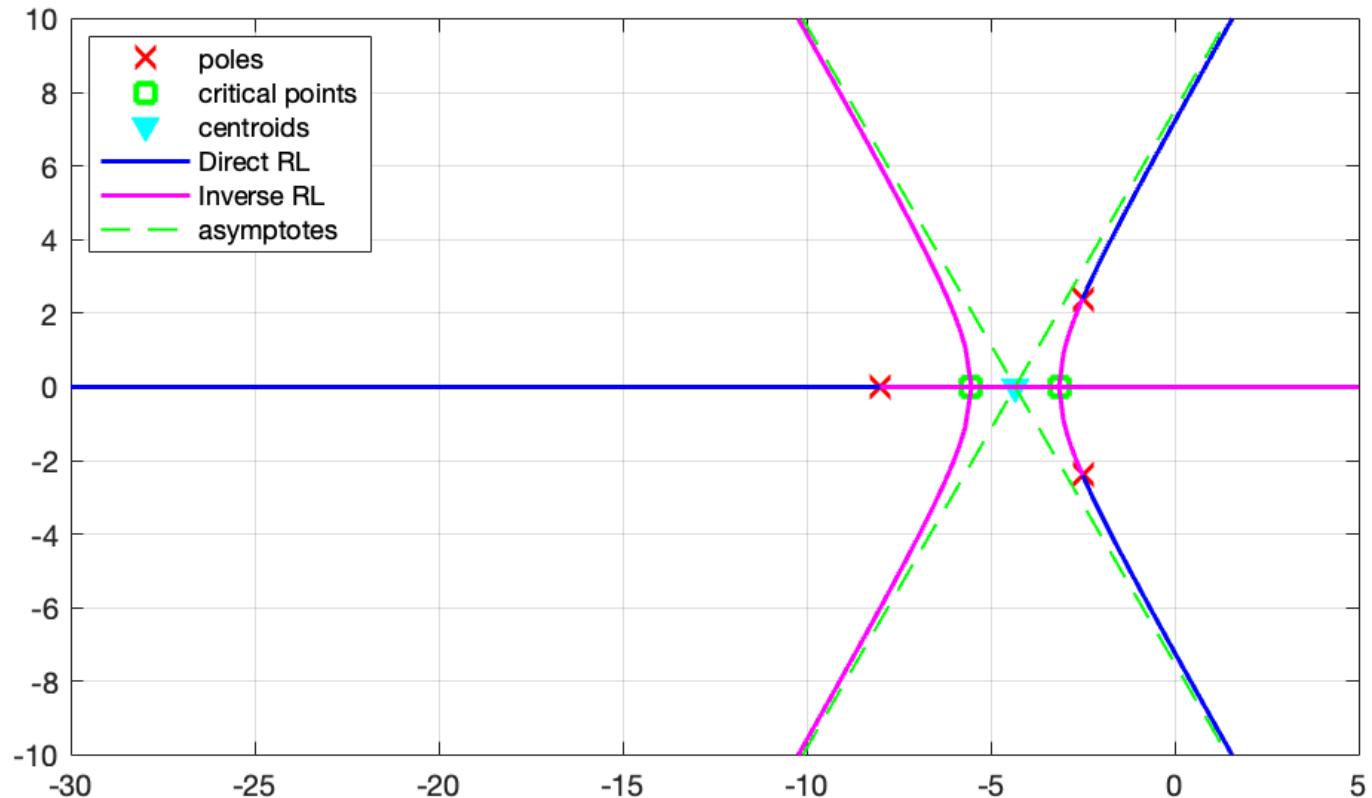
- One critical point belongs to the **Direct RL** and one to the **Inverse RL**

## Example 6

$$L(s) = \frac{1}{(s+8)(s^2 + 5s + 12)}$$

### Remarks:

- There are no open-loop zeros and all the open-loop poles are located in the left half-plane. The real open-loop pole is rather "distant" from the pair of complex-conjugate open-loop poles.
- Both the **Direct RL** and the **Inverse RL** have three asymptotes forming angles of  $120^\circ$  between them
- As shown by the **Direct RL**, increasing  $\varrho > 0$  makes the closed-loop system unstable
- Likewise, as shown by the **Inverse RL**, decreasing  $\varrho < 0$  makes the closed-loop system unstable



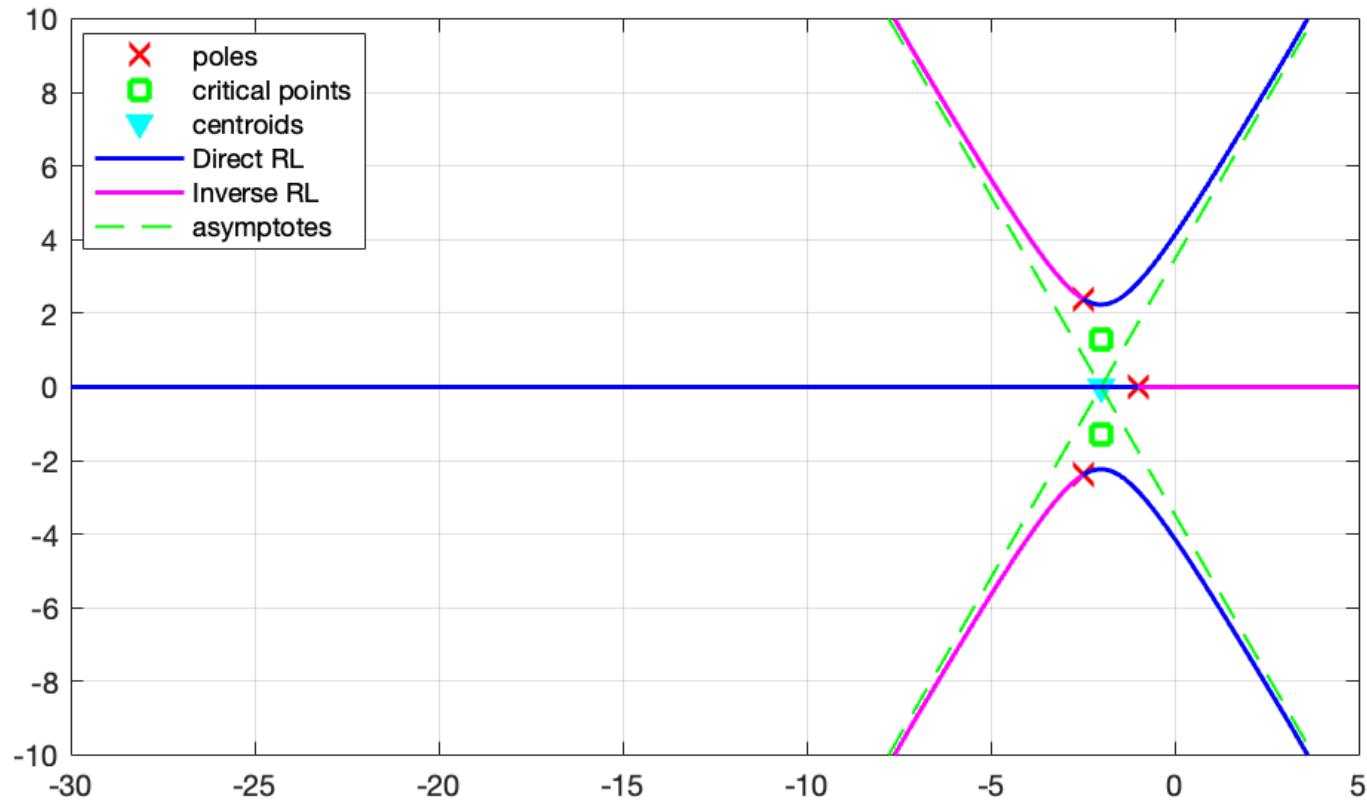
- Both critical points belong to the negative real axis as part of the **Inverse RL**

## Example 7

$$L(s) = \frac{1}{(s+1)(s^2 + 5s + 12)}$$

### Remarks:

- There are no open-loop zeros and all the open-loop poles are located in the left half-plane. Unlike Example 6, the real open-loop pole is rather "close" to the pair of complex-conjugate open-loop poles.
- Both the **Direct RL** and the **Inverse RL** have three asymptotes forming angles of  $120^\circ$  between them
- As shown by the **Direct RL**, increasing  $\varrho > 0$  makes the closed-loop system unstable
- Likewise, as shown by the **Inverse RL**, decreasing  $\varrho < 0$  makes the closed-loop system unstable



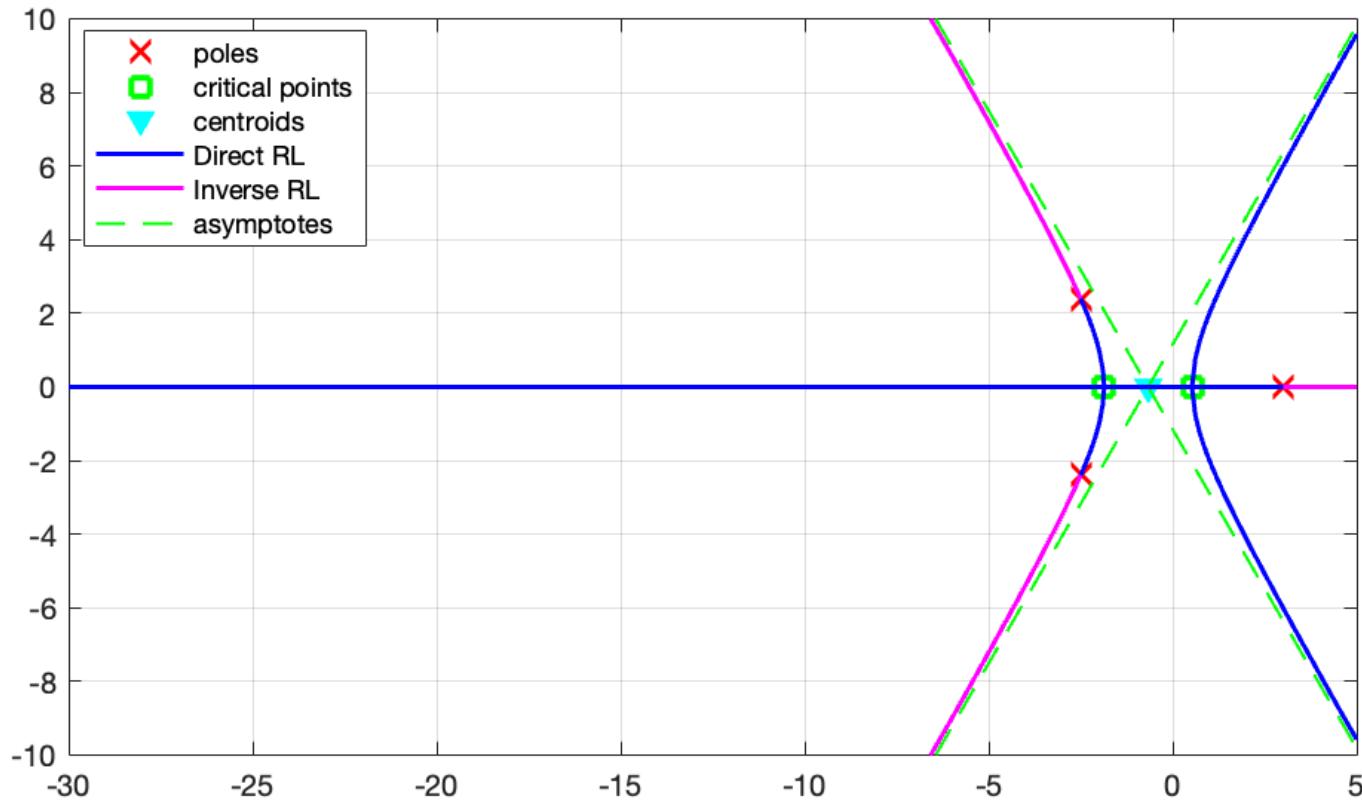
- Both critical points do **not** belong to the **RL** (neither the **Direct** nor the **Inverse**)

## Example 8

$$L(s) = \frac{1}{(s - 3)(s^2 + 5s + 12)}$$

### Remarks:

- There are no open-loop zeros and one of the open-loop poles is located in the right half-plane. The real open-loop pole is rather "distant" from the pair of complex-conjugate open-loop poles.
- Both the **Direct RL** and the **Inverse RL** have three asymptotes forming angles of  $120^\circ$  between them
- For any  $\varrho > 0$  the **Direct RL** shows that the closed-loop system is unstable
- Likewise, for any  $\varrho < 0$  the **Inverse RL** shows that the closed-loop system is unstable



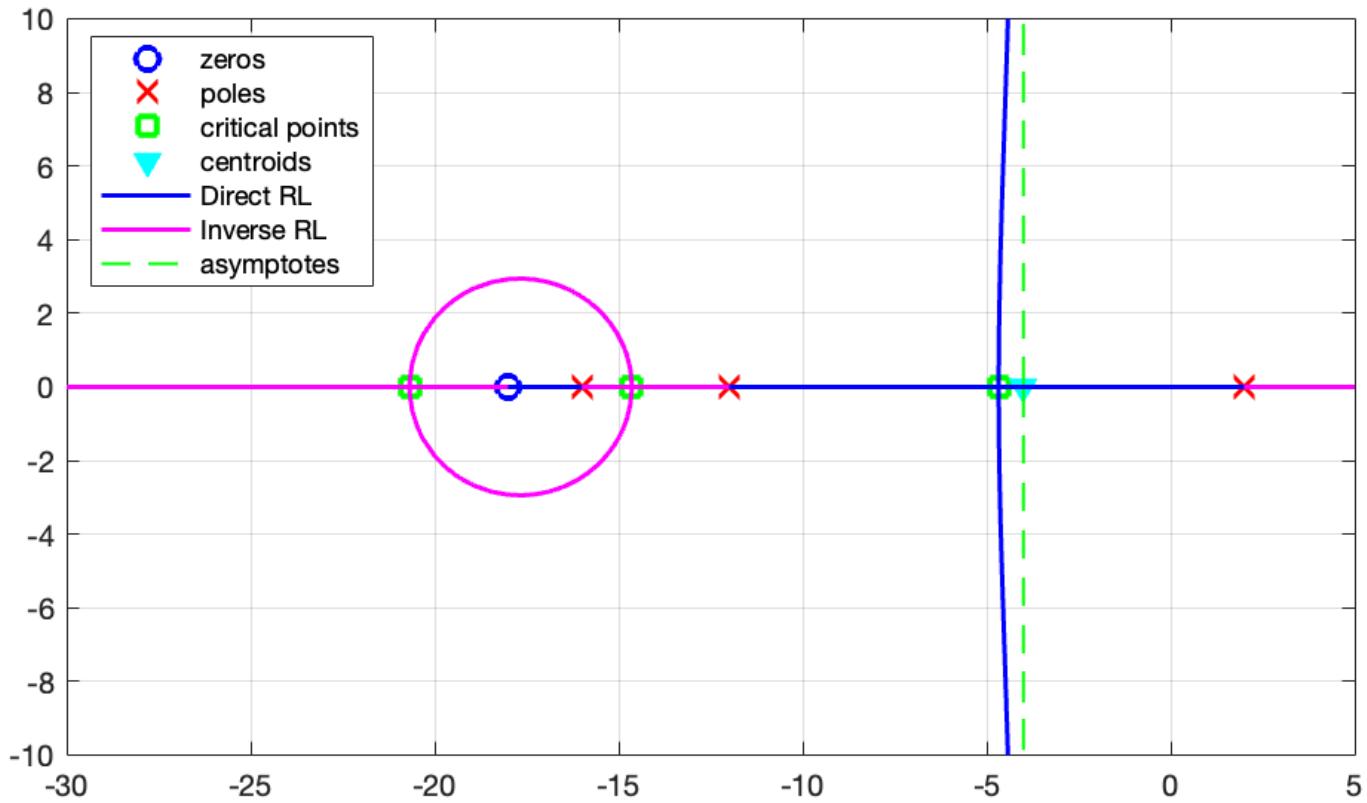
- The "form" of the RL is very similar to the one in Example 6
- But, both critical points belong to the **Direct RL**: one belongs to the negative real axis and the other to the positive real axis

## Example 9

$$L(s) = \frac{s + 18}{(s - 2)(s + 12)(s + 16)}$$

### Remarks:

- There is one open-loop zero located on the left of all open-loop poles, one of which is located in the right half-plane.
- In the region close to the zero and the negative poles, the **Inverse RL** has a form similar to the **Direct RL** in Example 1.
- Both the **Direct RL** and the **Inverse RL** have two asymptotes
- Increasing  $\varrho > 0$  makes the closed-loop system as. stable as shown by the **Direct RL**
- For any  $\varrho < 0$  the **Inverse RL** shows that the closed-loop system is unstable



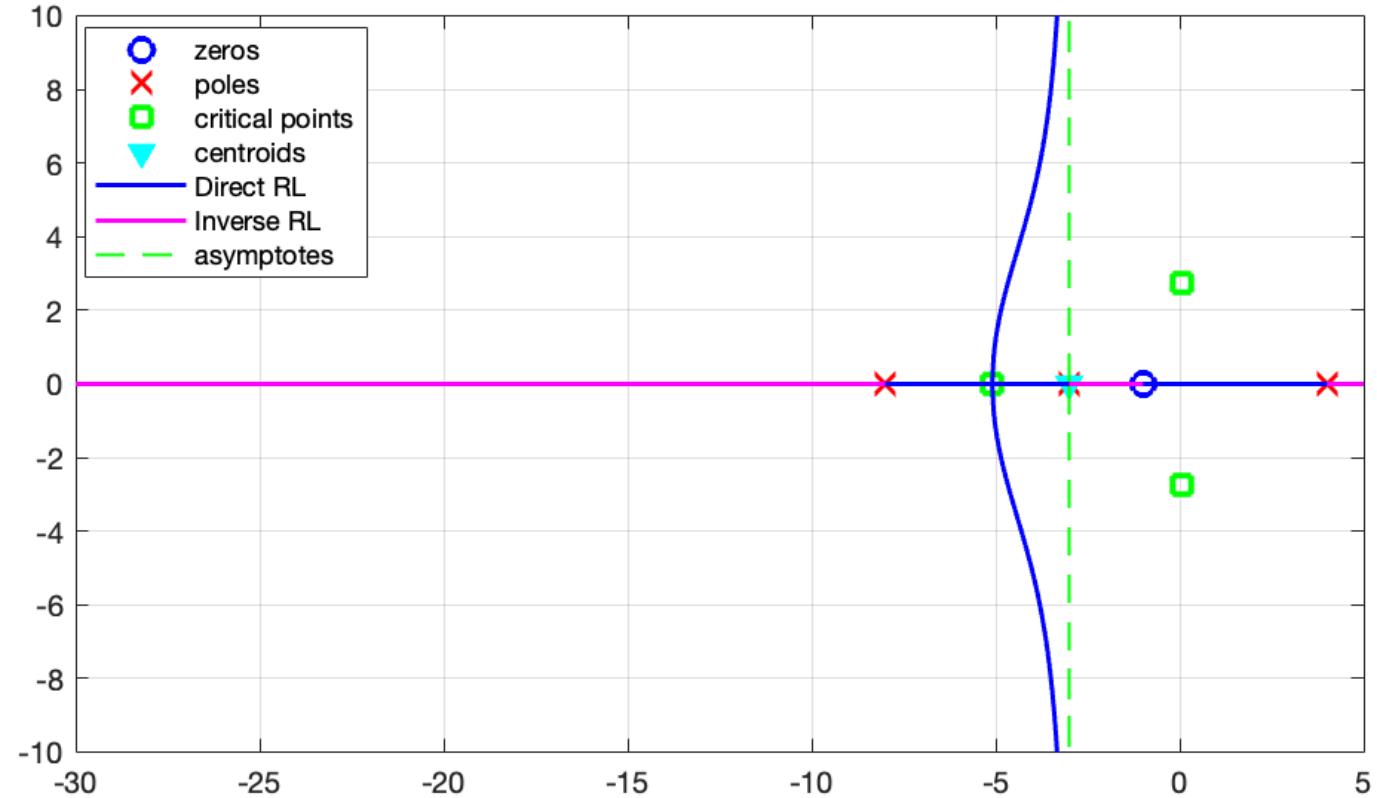
- One critical point belongs to the **Direct RL** and one to the **Inverse RL**
- The open-loop zero has a beneficial effect because its presence means having two asymptotes in the left half-plane

## Example 10

$$L(s) = \frac{s + 1}{(s - 4)(s + 3)(s + 8)}$$

### Remarks:

- There is one open-loop zero located on the right of the negative open-loop poles. There is one positive open-loop pole.
- Both the **Direct RL** and the **Inverse RL** have two asymptotes
- The zero "attracts" the centroid.
- Increasing  $\varrho > 0$  makes the closed-loop system as. stable as shown by the **Direct RL**
- For any  $\varrho < 0$  the **Inverse RL** shows that the closed-loop system is unstable



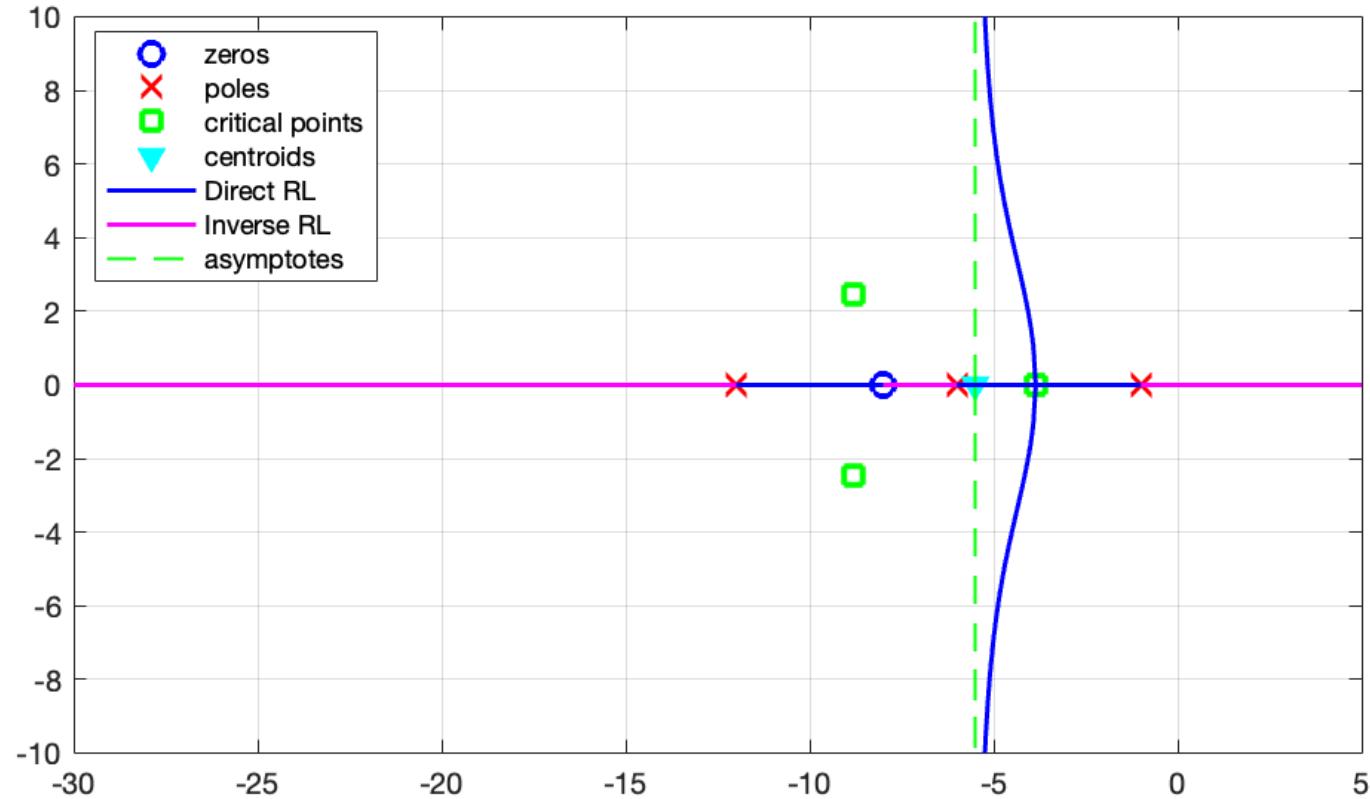
- One critical point belongs to the **Direct RL** but the other two do not belong to the **RL**

## Example 11

$$L(s) = \frac{s + 8}{(s + 1)(s + 6)(s + 12)}$$

### Remarks:

- All the open-loop zeros and poles are located in the left half-plane.
- Both the **Direct RL** and the **Inverse RL** have two asymptotes
- The zero "attracts" the centroid.
- For any  $\varrho > 0$  the **Direct RL** shows that the closed-loop system is as. stable
- Decreasing  $\varrho < 0$  makes the closed-loop system unstable as shown by the **Inverse RL**
- One critical point belongs to the **Direct RL** but the other two do not belong to the **RL**

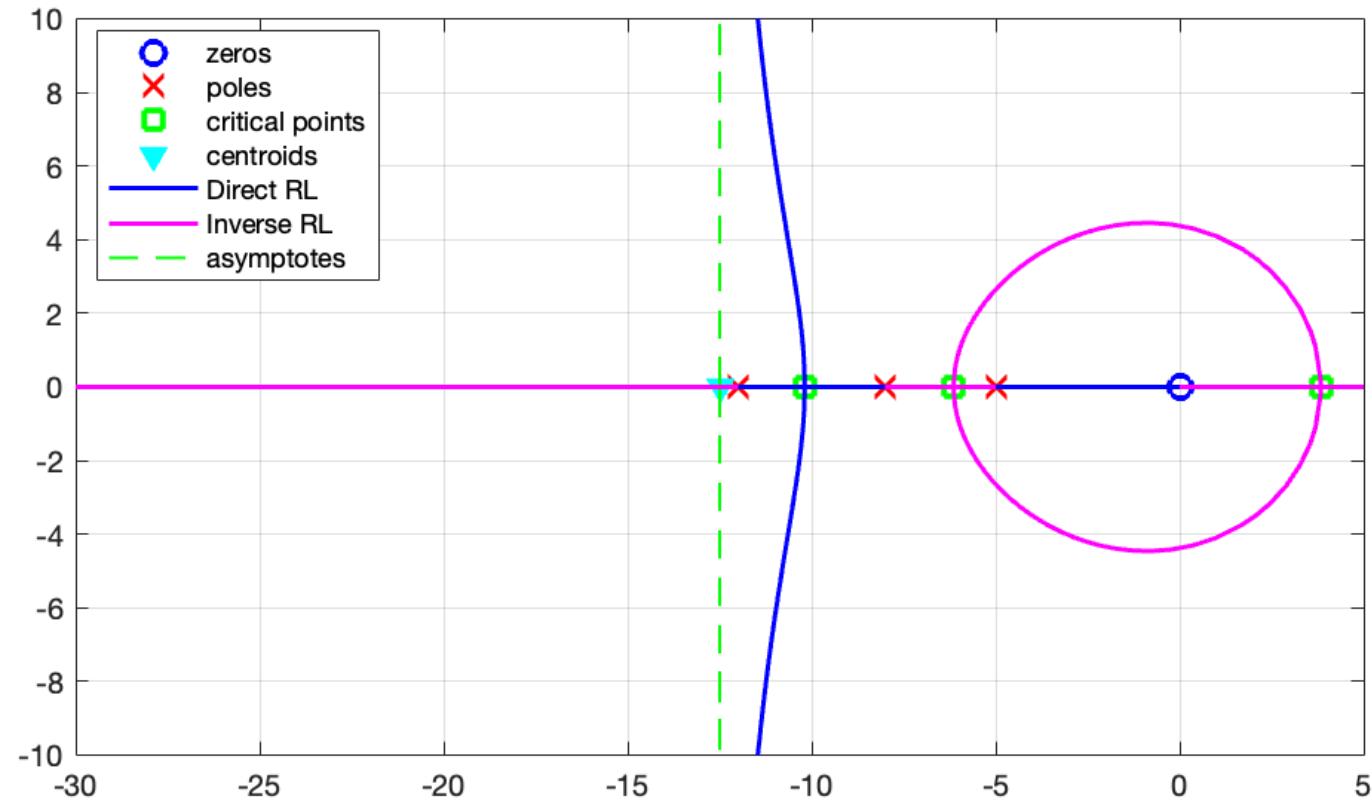


## Example 12

$$L(s) = \frac{s}{(s+5)(s+8)(s+12)}$$

### Remarks:

- The open-loop zero is located on the imaginary axis and all open-loop poles are located in the left half-plane.
- Both the **Direct RL** and the **Inverse RL** have two asymptotes
- The zero does not "attract" the centroid in this case because the pole in -5 is too close
- For any  $\varrho > 0$  the **Direct RL** shows that the closed-loop system is as. stable
- Decreasing  $\varrho < 0$  makes the closed-loop system unstable as shown by the **Inverse RL**



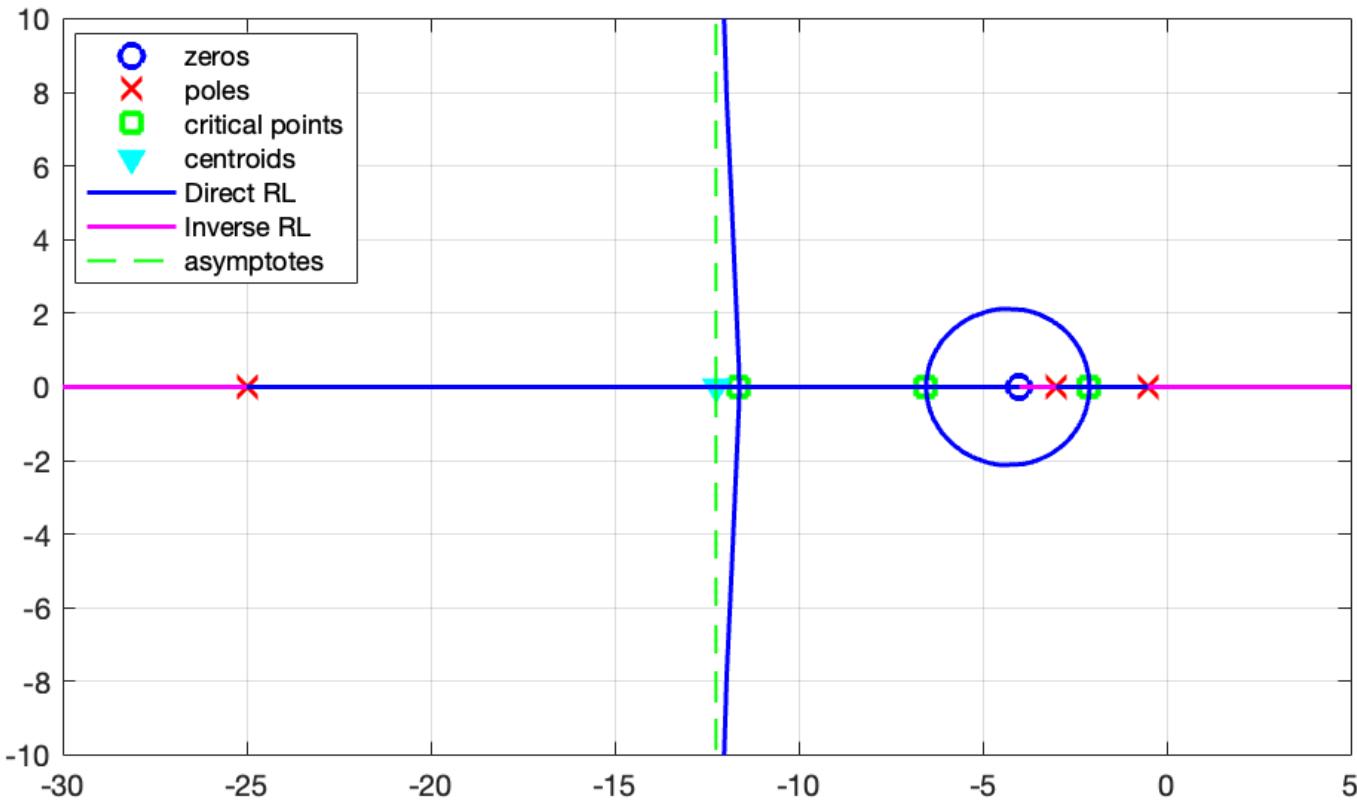
- One critical point belongs to the **Direct RL** and the other two belong to the **Inverse RL**

## Example 13

$$L(s) = \frac{s + 4}{(s + 1/2)(s + 3)(s + 25)}$$

### Remarks:

- All the open-loop zeros and poles are located in the left half-plane.
- The zero is very close with the pole in -3. Hence the zero does not "attract" the centroid.
- In the region of the zero and the two close negative poles, the form of the RL is similar to Example 1.
- Both the **Direct RL** and the **Inverse RL** have two asymptotes
- For any  $\varrho > 0$  the **Direct RL** shows that the closed-loop system is as. stable
- Decreasing  $\varrho < 0$  makes the closed-loop system unstable as shown by the **Inverse RL**



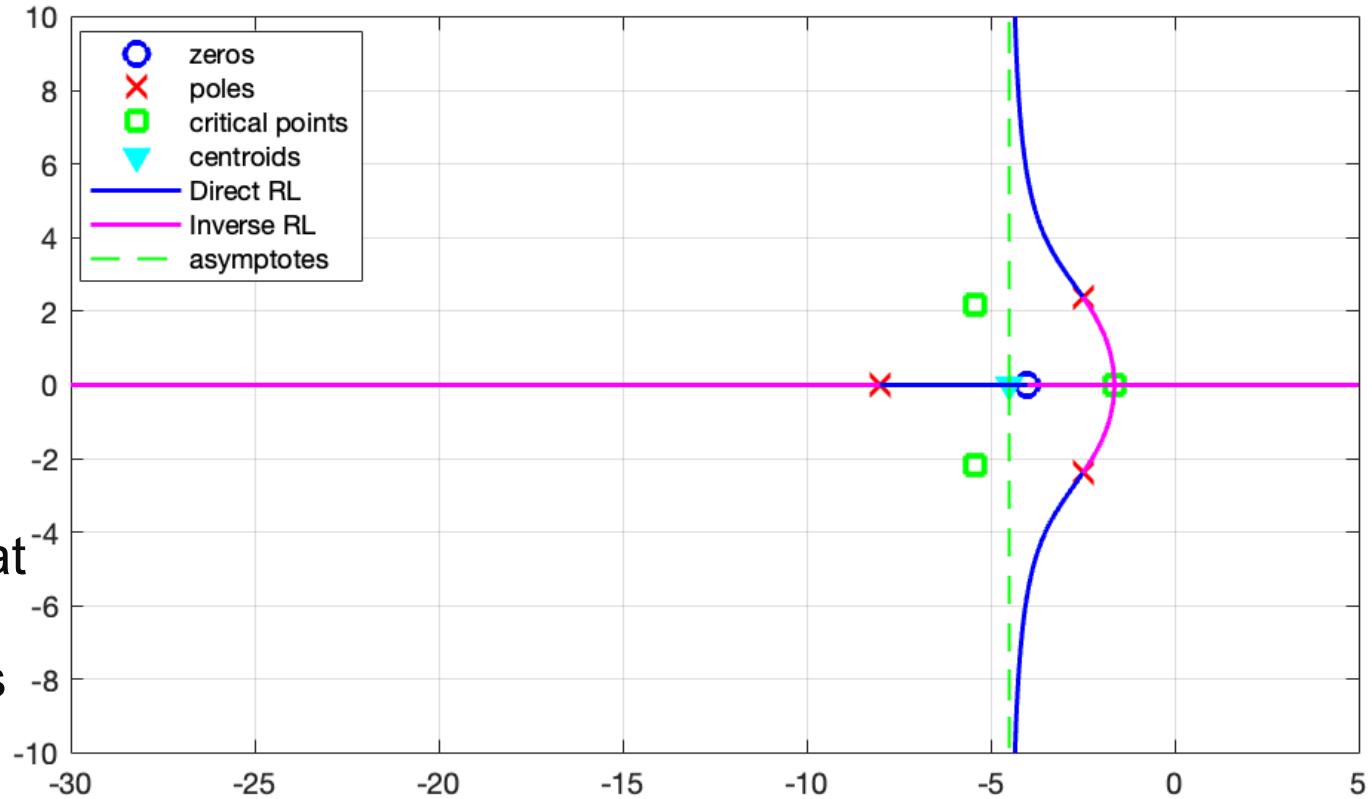
- All three critical points belong to the **Direct RL**

## Example 14

$$L(s) = \frac{s + 4}{(s + 8)(s^2 + 5s + 12)}$$

### Remarks:

- All the open-loop zeros and poles are located in the left half-plane.
- Compared to Example 6 (no open-loop zero), the form of the RL is modified by the zero.
- Both the **Direct RL** and the **Inverse RL** have two asymptotes
- For any  $\varrho > 0$  the **Direct RL** shows that the closed-loop system is as. stable.  
Compared with Example 6, the zero has a stabilising effect.
- Decreasing  $\varrho < 0$  makes the closed-loop system unstable as shown by the **Inverse RL**



- Only one critical point belongs to the RL