

Robotic Path Planning: Steering Limitations

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December 17, 2020

Overview

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Objective



Figure: Auriga- α robot with 2 trailers

Following Problems arise to control such robot,

- Trailer Motion Instability
- Inter-unit Collision
- Off Tracking

To overcome these issues, we can restrict the steering control of the driving unit during forward motion of the robot.

Kinematic Model for n-trailer robot

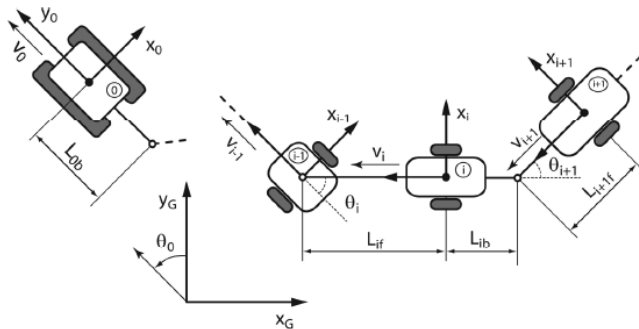


Figure: Kinematic Chain

- Total $n+1$ units for n -trailer system.
- L_f and L_b are the distances between hitch/axle of consecutive units.
- θ_i is relative angle of i^{th} trailer w.r.t $(i-1)^{\text{th}}$ unit.
- The control inputs for the system are, Speed v_0 and Curvature γ_0

Kinematic Model

- ϕ_i is heading of i^{th} trailer w.r.t global co-ordinate system.

$$\phi_i = \sum_{j=0}^i \theta_j$$

- ω_i is Angular Velocity of i^{th} unit,

$$\omega_i = \frac{d\theta_i}{dt}$$

- Ω_i is Absolute Angular Velocity is calculated as,

$$\Omega_i = \sum_{j=0}^i \omega_j$$

- γ_i is the curvature, and is given by

$$\gamma_i = \frac{d\phi_i}{ds_i} = \frac{\Omega_i}{v_i}$$

where s_i is the distance travelled and v_i is the speed.

Kinematic Model

$$\begin{bmatrix} v_{ix} \\ v_{iy} \end{bmatrix} = v_{i-1} \begin{bmatrix} \sin\theta_i \\ \cos\theta_i \end{bmatrix} + L_{i-1b}\Omega_{i-1} \begin{bmatrix} \cos\theta_i \\ -\sin\theta_i \end{bmatrix} + L_{if}\omega_i \begin{bmatrix} 1 \\ 0 \end{bmatrix} \quad (1)$$

Transversal Component $v_{ix} = 0$

Longitudinal Component $v_{iy} = v_i$

(Assuming Rolling without slipping motion)

$$v_i = v_{i-1}[\cos\theta_i - \gamma_{i-1}L_{i-1b}\sin\theta_i] \quad (2)$$

$$\omega_i = -v_{i-1} \left(\frac{\sin\theta_i}{L_{if}} + \frac{\gamma_{i-1}L_{i-1b}\cos\theta_i}{L_{if}} \right) \quad (3)$$

From Equation (2) and Equation (3), we obtain

$$\gamma_i = \begin{cases} \frac{\gamma_{i-1}L_{i-1b}\cos\theta_i + \sin\theta_i}{\gamma_{i-1}L_{i-1b}L_{if}\sin\theta_i - L_{if}\cos\theta_i} & L_{i-1b} \neq 0 \\ -\frac{\tan\theta_i}{L_{if}} & L_{i-1b} = 0 \end{cases} \quad (4)$$

- Observations: γ_i depends on θ_i and γ_{i-1}

Steady Response

We set v_{os} and γ_{os} as the constant set points for speed and curvature of the tractor unit.

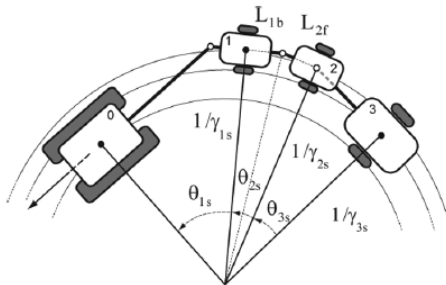


Figure: Constant Tractor set points

$$\gamma_{is} = \frac{\gamma_{i-1s}}{\sqrt{1 + \gamma_{i-1s}^2 (L_{i-1b}^2 - L_{if}^2)}} \quad (5)$$

Equilibrium Limit

$$\gamma_{is} = \frac{\gamma_{i-1s}}{\sqrt{1 + \gamma_{i-1s}^2 (L_{i-1b}^2 - L_{if}^2)}}$$

- This equilibrium exists only when γ_{is} is real valued,
 $L_{i-1b} > L_{if}$ or $\gamma_{i-1} \leq \frac{1}{\sqrt{L_{if}^2 - L_{i-1b}^2}}$
- In Forward Motion, this Equilibrium Limit is set as

$$\gamma_{i-1m_1} = \begin{cases} \frac{1}{\sqrt{L_{if}^2 - L_{i-1b}^2}} & L_{i-1b} < L_{if} \\ \infty & L_{i-1b} \geq L_{if} \end{cases} \quad (6)$$

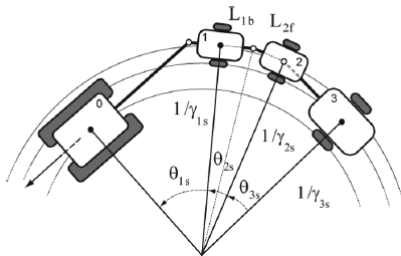


Figure: Constant Tractor set points

- Two equilibrium points for mechanical limit are geometrically possible,

$$\theta_{is}^{(1)} = -\arctan(\gamma_{i-1s}L_{i-1b}) - \arctan(\gamma_{is}L_{if})$$

$$\theta_{is}^{(2)} = \pi - \arctan(\gamma_{i-1s}L_{i-1b}) + \arctan(\gamma_{is}L_{if})$$

- During Forward motion, $\theta_{is}^{(2)}$ is unstable as it is near jack-knife position ($\pm\pi$).
- During reverse motion, $\theta_{is}^{(1)}$ is unstable as it is near jack-knife position

Mechanical Limit

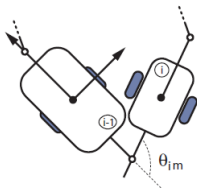


Figure: Physical Limit of Angle θ_i

- The joint between every pair of consecutive units (i-1) and i have an inherent mechanical limit θ_{im} .
- If this is surpassed, then there can be inter-unit collisions or Linkage breakage.
- Hence, the mechanical limit obtained from Mechanical Equilibrium must be imposed,

$$\gamma_{i-1m_2} = \frac{\sin\theta_{im}}{L_{if} + L_{i-1b}\cos\theta_{im}} \quad (7)$$

Backpropogated Limit

- The Steady state curvature is given by,

$$\gamma_{is} = \frac{\gamma_{i-1s}}{\sqrt{1 + \gamma_{i-1s}^2 (L_{i-1b}^2 - L_{if}^2)}}$$

- The $(i-1)^{\text{th}}$ unit should take into account the maximum steering limitations of the following i^{th} unit.

$$\gamma_{i-1m_3} = \frac{\gamma_{im}}{\sqrt{1 + \gamma_{im}^2 (L_{if}^2 - L_{i-1b}^2)}} \quad (8)$$

Among all the limits in Equation (6), Equation (7) and Equation (8), the most constraining one is chosen,

$$\gamma_{i-1m} = \min(\gamma_{i-1m_1}, \gamma_{i-1m_2}, \gamma_{i-1m_3}) \quad (9)$$

Algorithm 1: Tractor Curvature Limitation

```
for  $i = n$  to 1 do (Steady analysis)
    Compute equilibrium limit  $\gamma_{i-1 m_1}$  (17)
    Compute mechanical limit  $\gamma_{i-1 m_2}$  (18)
    Compute backpropagation limit  $\gamma_{i-1 m_3}$  (19)
     $\gamma_{i-1 m} = \min(\gamma_{i-1 m_1}, \gamma_{i-1 m_2}, \gamma_{i-1 m_3})$ 
if any hitch from 1 to  $n - 1$  is off-axle then
    repeat (Dynamic analysis)
         $\gamma_{0 m} \leftarrow (\gamma_{0 m} - \Delta\gamma)$ 
        Evaluate step response from  $\mp\gamma_{0 m}$  to  $\pm\gamma_{0 m}$ 
    until equilibrium is reached without inter-unit
    collision
Result:  $\gamma_{0 m}$ 
```

Implementation on Auriga- α

- The motion control system consists of
 - Path Generator
 - Path Tracker
 - Kinematic Model
 - Track Speed Controller
- The first 3 systems are implemented on CPU/Computer, whereas the Last system is implemented on onboard digital signal processor.
- Path Generator Generates Obstacle free list of points to be followed by Tractor.
- Path Tracker generates speed and curvature references in order to follow the path.
- Kinematic model and Track speed controller consists of two independent PID controllers which controls track speed.
- PID parameters provides overdamped track speed response so that overdamped response is obtained for curvature setpoints

Simulation

- The algorithm was implemented to determine the steering limitations for a 2 Trailer System with off-axle joints.
- The Axle Joints have maximum angle limits of $\theta_{1m} = 68^\circ$ and $\theta_{2m} = 43.6^\circ$
- The Steering limit obtained by using the Algorithm is 0.41m^{-1} .

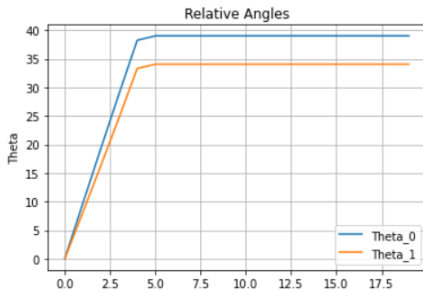


Figure: Response after Implementing Steering Limits

- The angle limits from above response is $\theta_{1m} = 39^\circ$ and $\theta_{2m} = 34^\circ$

Simulation

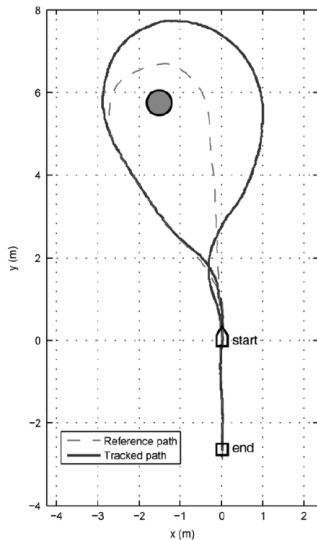


Figure: Path Tracking

Conclusion

- The Algorithm can be used for avoiding instability and inter-unit collision.
- The Algorithm relies on the existence of equilibrium of trailers.
- The Algorithm takes into account the type of joint/ hitch used for connecting the trailer units.
- The Steering limitations of the tractor can be imposed on Path Generator
- In such case, the tracking errors are negligible.

- 1] Jorge L. Martínez, Jesús Morales, Anthony Mandow, and Alfonso García-Cerezo "Steering Limitations for a Vehicle Pulling Passive Trailers". IEEE TRANSACTIONS ON CONTROL SYSTEMS TECHNOLOGY, Vol. 16, no. 4. pp 809-818, 2008
- 2] J.L. Martínez, A. Mandow, J. Morales, "Forward path tracking for Mobile Robots with several trailers". 5th IFAC/EURON Symposium on Intelligent Autonomous Vehicles Instituto Superior Técnico, Lisboa, Portugal, 2004.

Thank You