

The Longitudinal Impact of a Rigid Mass Against an Elastic Bar

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1 Problem Statement

Figure. 1 illustrates the longitudinal impact of a rigid mass against an elastic bar. The rigid body has a mass m_2 and an initial velocity v_0 whereas the elastic bar is fixed at $x = l$ and is free at the origin and has a mass m_1 , density ρ , length l with a cross-sectional area A and Young's modulus E . In the following, c is the wave speed.

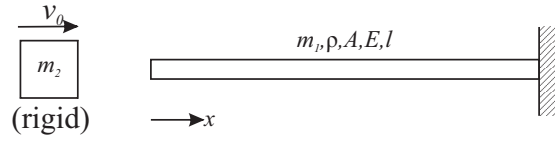


Figure 1: The problem geometry and coordinate system.

2 $0 < ct < 2l$ at $x = 0$

At time interval $0 < ct < 2l$, assuming that $t = 0$ is the time when the rigid mass hits the bar, a compressive forward wave $f(ct - x)$ is generated at $x = 0$ which takes $t = \frac{2l}{c}$ for the generated wave f_0 to return to $x = 0$. The subscript zero denotes the number given to the generated wave. The displacement, u , and strain, ϵ , for this time interval are given here for convenience as it is used later in the subsequent sections.

$$\begin{aligned} u_0(x, t) &= f_0 = \frac{lv_0}{Mc} \left(1 - e^{-\frac{M}{l}ct} \right) \\ \epsilon &= -\frac{v_0}{c} e^{-\frac{M}{l}ct} \end{aligned} \quad (1)$$

where

$$M = \frac{\rho Al}{m_2} = \frac{EA l}{m_2 c^2} \quad (2)$$

3 $l < ct < 3l$ at $x = l$

As soon as the compressive backward wave $f_0(ct - x)$ arrives at the fixed boundary ($x = l$), it reflects as a compressive backward wave $g_1(ct + x)$. g_1 has a latency of $\frac{l}{c}$; therefore, this backward wave is written as $g_1(ct + x - 2l)$. Writing and simplifying the relevant equations results in

$$g_1 = \frac{v_0 l}{cM} \left(e^{-\frac{M}{l}(ct + x - 2l)} - 1 \right) \quad (3)$$

The subscript one represents the generated wave number.

4 $2l < ct < 4l$ at $x = 0$

At this time interval, both the forward and backward wave contribute to the displacement and it can be written as

$$\begin{aligned} u_1(x, t) &= f_1(ct - x - 2l) + g_1(ct + x - 2l) \\ &= f_1(\xi_1) + g_1(\xi_2) \end{aligned} \quad (4)$$

in which

$$\begin{cases} \xi_1 = ct - x - 2l \\ \xi_2 = ct + x - 2l \end{cases}$$

In Eq. 4, g_1 is substituted from Eq. 3 and f_1 is unknown.

4.1 Determination of f_1

One can write Newton's second law for Figure. 1 for $2l < ct < 4l$ at $x = 0$

$$\begin{aligned} -m_2 \frac{\partial^2 u(x, t)}{\partial t^2} &= -EA \frac{\partial u(x, t)}{\partial x} \\ m_2 c^2 (f_1''(\xi_1) + g_1''(\xi_2)) &= EA (g_1'(\xi_2) - f_1'(\xi_1)) \end{aligned}$$

where the chain rule is utilized in calculating the derivatives, *i.e.*, $\frac{\partial f_1}{\partial \xi_1} \frac{\partial \xi_1}{\partial t}$.

Because all the above calculations are done at the free end of the bar ($x = 0$), ξ_1 and ξ_2 can be replaced by $\xi = ct - 2l$.

$$\begin{aligned} m_2 c^2 (f_1''(\xi) + g_1''(\xi)) &= EA (g_1'(\xi) - f_1'(\xi)) \\ f_1''(\xi) + g_1''(\xi) &= \frac{M}{l} (g_1'(\xi) - f_1'(\xi)) \end{aligned} \quad (5)$$

M can be found from Eq. 2.

g_1 and its derivatives in Eq. 5 can be calculated from Eq. 3.

$$\begin{aligned} g_1'(ct + x - 2l) &= -\frac{v_0}{c} e^{-\frac{M}{l}(ct + x - 2l)} \\ g_1'(\xi)_{\{x=0\}} &= -\frac{v_0}{c} e^{-\frac{M}{l}\xi} \\ g_1''(ct + x - 2l) &= \frac{Mv_0}{lc} e^{-\frac{M}{l}(ct + x - 2l)} \\ g_1''(\xi)_{\{x=0\}} &= \frac{Mv_0}{lc} e^{-\frac{M}{l}\xi} \end{aligned}$$

One should note that the above derivatives are taking w.r.t x .

Replacing $g_1'(\xi)$ and $g_1''(\xi)$ from above set of equations and using the `dsolve` command from Maple, the solution to Eq. 5 is

$$f_1(\xi) = \frac{(2Mv_0\xi - Bcl + 2lv_0)}{Mc} e^{-\frac{M}{l}\xi} + D \quad (6)$$

where B and D are constants and can be determined from initial conditions. In order to find B and D , we use the continuity principle of displacements and velocity where both the initial displacement and velocity in Eq. 4 should have the same value of u and v at $ct = 2l$ in Eq. 1, that is

For the displacement:

$$\begin{aligned} u_{1\{x=0, ct=2l\}} &= u_{0\{x=0, ct=2l\}} \\ f_{1\{x=0, ct=2l\}} + g_{1\{x=0, ct=2l\}} &= f_{0\{x=0, ct=2l\}} \end{aligned} \quad (7)$$

For the velocity:

$$\begin{aligned} v_{1\{x=0, ct=2l\}} &= v_{0\{x=0, ct=2l\}} \\ \frac{\partial}{\partial t} f_{1\{x=0, ct=2l\}} + \frac{\partial}{\partial t} g_{1\{x=0, ct=2l\}} &= \frac{\partial}{\partial t} f_{0\{x=0, ct=2l\}} \end{aligned} \quad (8)$$

Solving Eq. 7 and Eq. 8 simultaneously

$$B = \frac{v_0}{c} (e^{-2M} + 1) \quad , \quad D = 0$$

As a result

$$f_1(\xi) = \frac{v_0}{Mc} (2M\xi - le^{-2M} + l) e^{-\frac{M}{l}\xi} \quad (9)$$

5 Displacement and Strain Plots

The displacement for $2l < ct < 4l$ can be found by plugging-in Eq. 3 and Eq. 9 into Eq. 4.

$$u_1(x, t) = \frac{v_0}{Mc} \left\{ [2M(ct - 2l - x) - l(e^{-2M} + 1) + 2l] e^{-\frac{M}{l}(ct - 2l - x)} + l \left(e^{-\frac{M}{l}(ct - 2l + x)} - 1 \right) \right\} \quad (10)$$

Figure. 2 shows the piecewise combination of Eq. 1 and Eq. 10 for $0 < ct < 4l$ for $c = 1$, $l = 1$, $M = 1$ at the free side of the elastic bar ($x = 0$). As expected, the displacement plot is continuous.

The strain can be calculated by taking the derivative of displacements from Eq. 1 and Eq. 10 w.r.t x .

$$\begin{cases} \epsilon_0 = \frac{v_0}{c} e^{-\frac{M}{l}ct} & , 0 < ct < 2l \\ \epsilon_1 = \frac{v_0}{c} \left\{ \left(\frac{2M(ct - 2l - x) - l(e^{-2M} + 1) + 2l}{l} - 2 \right) e^{-\frac{M}{l}(ct - 2l - x)} - e^{-\frac{M}{l}(ct - 2l + x)} \right\} & , 2l < ct < 4l \end{cases} \quad (11)$$

Eq. 11 is plotted in Figure. 3. As it is seen, the discontinuity in the strain plot occurs at $t = 2l$ where the strain jumps to higher but negative value. It is also evident that for $0 < ct < 2l$, the strain does not become zero as it appears in form of an exponential function (see Eq. 11₁), while the strain and subsequently the stress (due to linear elastic assumption between stress and strain) vanishes at around $3l$. To find the corresponding time, we may solve Eq. 11₂ for t .

$$t = \frac{1}{2} \frac{4Ml + 2Mx + le^{-2M} + l + le^{-\frac{2M}{l}x}}{Mc}$$

If we remove x from above equation

$$t = \frac{1}{2} \frac{4Ml + le^{-2M} + 2l}{Mc} \quad (12)$$

and for $c = 1$, $l = 1$, $M = 1$, the time at which stress vanishes is 3.068s.

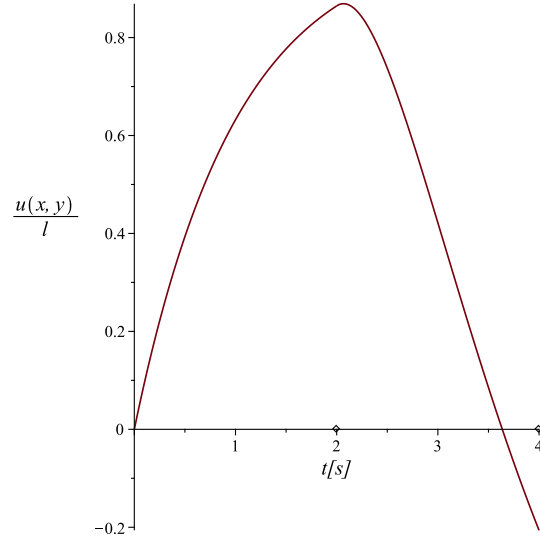


Figure 2: Normalized displacement against time for $c = 1$, $l = 1$, $M = 1$ at the free side of the elastic bar ($x = 0$).

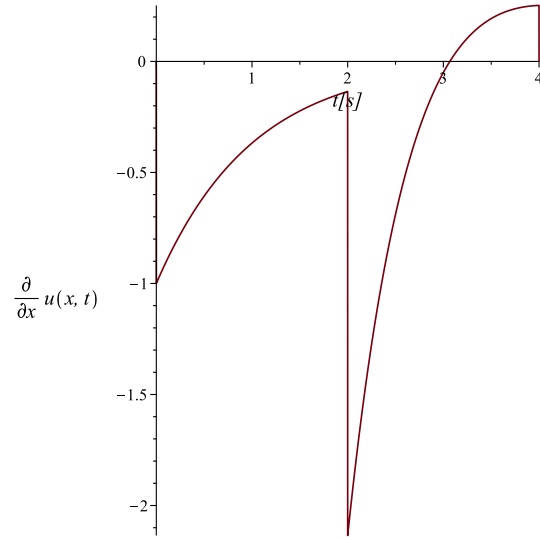


Figure 3: Strain against time for $c = 1$, $l = 1$, $M = 1$ at the free side of the elastic bar ($x = 0$).