

Thermal Physics - I

Development of Kinetic Theory:

Anonymous theory: 1761. J. Black: observed same temperature for ice & water & conjectured about "latent" heat.

1797 T. Watt: converted heat \rightarrow work as steam engine

1797 B. Thompson, C. Rumford: work \rightarrow heat

Caloric fluid theory: 1783 A. Lavoisier: conjectured heat as invisible fluid.

1824 S. Carnot: Carnot engine \rightarrow Thermochemistry.

Kinetic theory:

1738 D. Bernoulli: molecular theory of fluid.

1847 J.P. Joule, J. von Mayer, H. von Helmholtz: heat & work equivalent form of energy.

1850 R. Clausius: (a) showed that "caloric fluid theory" is reconciled in "kinetic theory", (b) First law of Thermodynamics, (c) defined "entropy".

1854 Thomson or Lord Kelvin \rightarrow Thermodynamics

Experimental gas laws:

1662 Boyle's law: $P \propto V^{-1}$

1802 Charles' law: $V \propto T$

1802 Gay-Lussac's law:

International scale of temperature

Lord Kelvin introduced lowest temperature independent of property of substance

absolute zero -273°C or 0K

ice point 0°C or 273K

steam point 100°C or 373K

Sulphur point 444.6°C

Antimony point 630.5°C

Silver point 960.8°C

Gold point 1063.8°C

Human body temperature $98.1^{\circ}\text{F} \leftrightarrow 36.9^{\circ}\text{C}$ (remember, least count is 0.1°C)
relation between C, F, R (Reumer scale)

$$\frac{C}{5} = \frac{F - 32}{9} = \frac{R}{4}.$$

Basics of Thermometry

- Liquid Thermometer: Hg in glass, 6's max-min, Beckmann, Index
- Gas Thermometer: Callendar's Air, Hydrogen
- Metal Thermometer: Platinum resistance,
- Thermoelectric Thermometer: thermocouple generates e.m.f.
- Radiation Thermometer: $e = at + bt^2$

principle of thermometry: let A = thermometric property
for uniform variation of A with temperature T

$$A_T = A_0 (1 + \alpha T), \quad A_0 = \text{value of } A \text{ at } 0^{\circ}\text{C}$$
$$A_{100} = \text{value of } A \text{ at } 100^{\circ}\text{C}.$$

$$A_{100} = A_0 (1 + 100\alpha)$$

$$\therefore T = \frac{100}{A_{100} - A_0} \frac{A_T - A_0}{A_0}$$

What is A ? If you take simple Hg-in-glass thermometer
then $A = \underline{\text{length}}$ of Hg column. For Pt-resistance thermometer
 $A = \underline{\text{resistance}}$ R of Pt. For constant volume gas thermometer
 $A = \underline{\text{pressure}}$ of air.

Hg in glass thermometer

- pros (a) liquid range $[-38.87^{\circ}\text{C}, 356^{\circ}\text{C}]$
(b) low thermal capacity & high conductivity
(c) uniform expansion, (d) shining liquid (easy to see).
- cons (a) high specific gravity (less sensitive)
(b) high surface tension (motion is jerky)

unsuitable for low T.

Alcohol thermometer

- pros
- liquid range $[-112^{\circ}\text{C}, 78^{\circ}\text{C}]$
 - high coefficient of expansion (more sensitive)
 - low surface tension (motion is smooth)
- cons
- unsuitable for high temperature.

Gas thermometer

- Very high coefficient of expansion (more sensitive)
- low thermal capacity, low specific gravity
- constant volume / constant pressure thermometer.

Platinum Resistance thermometer : Resistance smooth in range $[-200^{\circ}\text{C}, 1200^{\circ}\text{C}]$
 least count = 0.01^{\circ}\text{C} $R_t = R_0(1 + \alpha T)$

Low temperature Thermometry

- 0°C to $630^{\circ}\text{C} \rightarrow$ all
- -2°C to $0^{\circ}\text{C} \rightarrow$ liquid, gas, resistance, thermocouple, vapour pressure.
- -272°C to $-2^{\circ}\text{C} \rightarrow$ resistance, vapour pressure, magnetic
- below $-272^{\circ}\text{C} \rightarrow$ magnetic

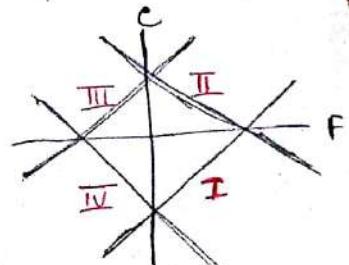
High temperature Thermometry

- 630°C to $1000^{\circ}\text{C} \rightarrow$ liquid, gas, resistance, thermocouple.
- 1000°C to $1600^{\circ}\text{C} \rightarrow$ gas, thermocouple.
- 1600°C to $3000^{\circ}\text{C} \rightarrow$ thermocouple, pyrometer.

HW 1. At what temperature do the Fahrenheit & Celsius scale give the same reading?

2. In figure beside, which line represents $\frac{G}{5} = \frac{F - 32}{9}$?

3. In a constant volume gas thermometer, pressure of air at 0°C is 80 cm & at 100°C is 109.3 cm. Calculate the temperature of a hot bath, in which when the thermometer is immersed shows a pressure 100 cm.



Calorimetry Definition of Calorie (heat) \Rightarrow quantity of heat needed to raise temperature of 1 gm of water by 1°C . But heat $5^{\circ}\text{C} \rightarrow 6^{\circ}\text{C} \neq 80^{\circ}\text{C} \rightarrow 81^{\circ}\text{C}$.

Mean calorie heat $[0^{\circ}\text{C} \rightarrow 100^{\circ}\text{C}] / 100$.

15°C Calorie heat $14.5^{\circ}\text{C} \rightarrow 15.5^{\circ}\text{C}$

B.Th.U. quantity of heat needed to raise temperature of 1 lb of water by 1°F .

1 B.Th.U. = 252 calz.

1 C.H.U. = 153.6 calz.
(centigrad heat unit)

$$H = m s \frac{T}{\text{temperature}} = c \frac{T}{\text{heat}}$$

↓
mass
of substance ↓
specific
heat ↓
thermal
capacity

Basic principle of Calorimetry conservation of heat energy or sum total of (body + heat bath) is conserved.

Specific heat of solid: Method of mixture, Copper block calorimeter, Electrical method (Lindemann calorimeter), Latent heat (Bunsen ice calorimeter).

Specific heat of liquid: Method of mixture, Bunsen ice calorimeter, Method of cooling, Nernst calorimeter, continuous flow calorimeter.

Specific heat of gas: # why constant pressure or volume?

$c = \frac{dq}{dt}$ \rightarrow suppose system expands \rightarrow temperature falls to prevent, dq heat added, so $dT = 0$.

$c = \frac{dq}{0} \rightarrow \infty$
 \rightarrow suppose system compresses \rightarrow temperature rise dT without the need of dq $\therefore dq = 0$
 $c = \frac{0}{dT} = 0$.

$$\therefore C_p = \left(\frac{dQ}{dT}\right)_p, \quad C_v = \left(\frac{dQ}{dT}\right)_v$$

definition: at $P = \text{constant}$, increase temperature of 1 gm of gas through 1°C . If 1 gm molecule of gas is taken \rightarrow molar specific heat. $C_p = MC_p$

$C_p > C_v$? # heat transfer at $V = \text{constant}$, $dV = 0$. $dW = PdV = 0$
no work done by the gas.

$C_p - C_v = \frac{R}{J}$ So heat supplied transformed to thermal energy

(heat unit) # heat transfer at $P = \text{constant}$, $dW = PdV \neq 0$.

$C_p - C_v = R$ so heat supplied \rightarrow thermal energy + external work.

Isothermal transformation : $T = \text{constant}$ $PV = RT = \text{constant}$

Adiabatic transformation : $Q = \text{constant}$, $dQ = C_v dT + PdV$. — ①

Now from $PV = RT$, $PdV + Vdp = RdT = (C_p - C_v)dT$

put dT in ① : $dQ = C_v \frac{PdV + Vdp}{C_p - C_v} + PdV = 0$ (as $Q = \text{constant}$)

$$\therefore C_v Vdp + C_p PdV = 0$$

$$\therefore \frac{C_p}{C_v} \frac{dV}{V} = - \frac{dp}{p}, \quad (\text{put } \alpha = \frac{C_p}{C_v})$$

Integrating both sides, $\therefore \ln V = - \ln p + \ln C$.

$$\boxed{PV^\gamma = \text{constant}}$$

Corollary: PT relation: put $PV = RT$, $P\left(\frac{RT}{P}\right)^\gamma = C \Rightarrow P^{1-\gamma} T^\gamma = \text{constant}$

VT relation: put $PV = RT$, $\frac{RT}{V} V^\gamma = C \Rightarrow V^{\gamma-1} T = \text{constant}$



Suppose dry air is enclosed in a cylinder fitted with piston. Initial temperature of air is 30°C . Find the change in temperature if the gas is compressed (a) slowly, (b) suddenly to one-half of its volume? What'll be the pressure?

(a) piston moves inwards \rightarrow gas compresses
 \downarrow



no change \leftarrow heat goes to \leftarrow temperature rises
 in surrounding by
 temperature conducting wall
 (isothermal) slowly

$$T = \underline{30^\circ C}$$

(b) Temperature rise rate \ggg heat diffusion rate to surrounding
 adiabatic process \rightarrow no Q transfer.

$$\text{So } P_1 V_1^{\gamma} = P_2 V_2^{\gamma} \text{ & given } V_2 = \frac{V_1}{2}$$

$$\therefore P_2 = 2^{\gamma} P_1 = 2^{1.4} P_1 = \underline{2.6 \text{ atm}} \text{ when } P_1 = \underline{1 \text{ atm.}}$$

$$\text{Compare with isothermal, } P_2 = 2P_1 = \underline{2 \text{ atm.}}$$

$$\text{Again from TV relation, } T_1 V_1^{\gamma-1} = T_2 V_2^{\gamma-1}$$

$$\therefore T_2 = T_1 \left(\frac{V_1}{V_2} \right)^{\gamma-1} = 2^{0.4} T_1 = 1.319 \times (273 + 30) K \\ = 399.7 K = (399.7 - 273)^\circ C = \underline{127^\circ C}$$

HW 4. Air in a Wilson's cloud chamber at $20^\circ C$ is abruptly expanded to 1.4 times its initial volume. Calculate the final temperature.
 Given $\gamma = 1.4$.

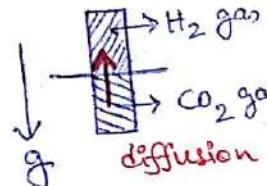
5. At 10^6 dynes/cm² pressure, a gas ($\gamma = 1.4$) expands isothermally until its volume is double of the initial volume. Then it adiabatically expands until its volume is redoubled. Calculate the final pressure of the gas.

Kinetic Theory of Gases (K.T.)

Rule of thumb: Every material (solid, liquid, gas, plasma, intermediate phases) are made of atoms. They "may" attract or repel & form molecules of liquid or be restricted in definite shape of solid by huge cohesive force.

Experimental hints in favour of K.T.

1. Diffusion and Solution:



when slowly poured Alcohol over water, it spreads throughout.

2. Expansion of substance with heat.: atoms tend to move away.
3. Phenomena of evaporation & vapour pressure.
4. Brownian motion. 1827 R. Brown + incessant motion of pollen on water.

Basic assumptions & postulates of K.T.

1. A gas consists of large number of identical atoms, which are rigid, elastic & equal mass objects.
2. Atoms are in chaos \rightarrow motion is fully irregular & spans in all three directions.
3. Inevitably the gas molecules collide with each other & surface of container (wall, sphere, cylinder). Total K.E. remains constant, but velocity of each atom continuously changes both in magnitude & direction. In evolving state (intermediate) density in a volume element will change but in steady state, collisions do not affect the density.
4. In between two successive collisions, molecules move in straight line following Newton's law.
5. Collisions are perfectly elastic i.e. no force of attraction/repulsion ($P.E. = 0$), energy is fully kinetic.

6. Atoms are "point" mass, meaning, their total volume <<< volume of the container.

Concept of solid angle

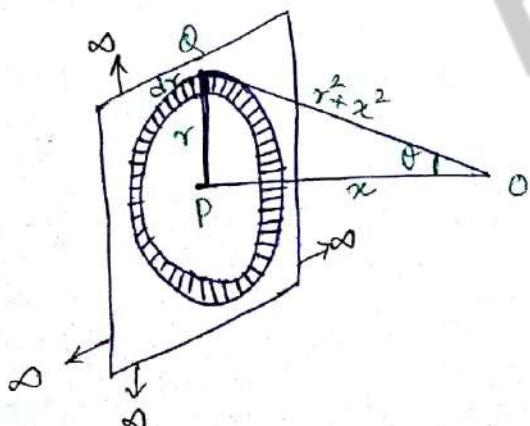
Solid angle subtended by an area at a point is defined as the area intercepted by the cone on a unit sphere (radius = 1) with its centre at the apex of the cone.

If dS is an area that makes a solid angle $d\omega$ at origin O at a distance $OO' = r$, then from similar figures,

$$\frac{d\omega}{1^2} = \frac{dS \cos\theta}{r^2} \quad \therefore d\omega = \frac{dS \cos\theta}{r^2}$$

Unit of solid angle = steradian.

- CW 1. Calculate the solid angle (α) subtended by an infinite plain at a point in front of it, (b) hemisphere and (c) full sphere at its center.



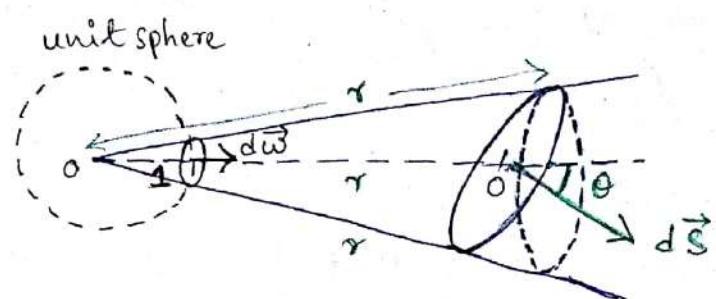
Consider the annular ring, r distance apart from P & thick dr .

$$\text{Area of this ring} = \pi(r+dr)^2 - \pi r^2 \\ = 2\pi r dr$$

(N.B. we throw $o(dr)$ term in limit $dr \rightarrow 0$)

So solid angle subtended by that circular annulus

$$d\omega = \frac{dS \cos\theta}{r^2} = \frac{2\pi r dr \cos\theta}{r^2}$$

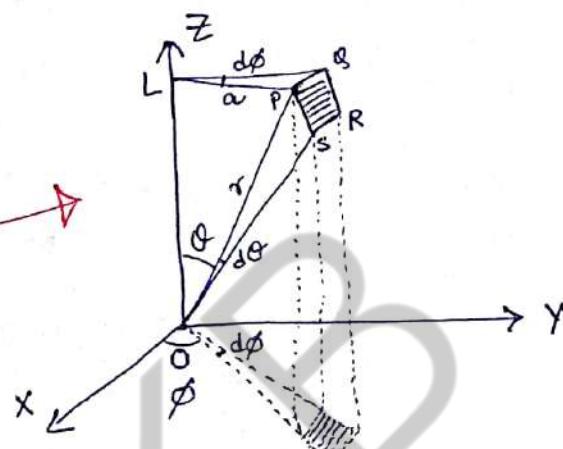
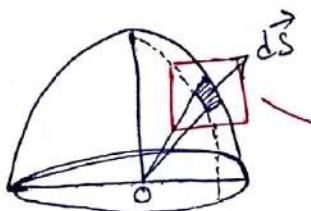


Infinite plain meaning θ going from 0 to $\pi/2$.

$$\begin{aligned}\therefore \omega &= \int_{\theta=0}^{\pi/2} \frac{2\pi r dr \cos\theta}{r^2 + x^2} \\ &= 2\pi \int_{\theta=0}^{\pi/2} \frac{x \tan\theta \sec^2\theta d\theta \cos\theta}{x^2 \sec^2\theta} \\ &= 2\pi \int_0^{\pi/2} \sin\theta d\theta = \underline{2\pi}.\end{aligned}$$

$$\begin{aligned}[r &= x \tan\theta] \\ dr &= x \sec^2\theta d\theta \\ &\& x^2 + r^2 = x^2 \sec^2\theta\end{aligned}$$

Hemisphere



$$LP = a \\ OP = r$$

$$\frac{a}{r} = \sin\theta$$

$$\therefore PQ = ad\phi$$

$$PS = r d\theta$$

$$\therefore dS = \text{area } PQRS = ad\phi \times r d\theta = r^2 \sin\theta d\theta d\phi.$$

$$\text{So } dw \text{ at point } O = \frac{r^2 \sin\theta d\theta d\phi}{r^2} \times \cos\theta = \sin\theta d\theta d\phi.$$

$$\text{So solid angle subtended} = \int dw = \int_{\theta=0}^{\pi/2} \int_{\phi=0}^{2\pi} \sin\theta d\theta d\phi = \underline{2\pi}.$$

Full sphere

$$\text{solid angle subtended} = \int_{\theta=0}^{\pi} \int_{\phi=0}^{2\pi} \sin\theta d\theta d\phi = \underline{4\pi}.$$

We will find out now pressure exerted by a perfect gas

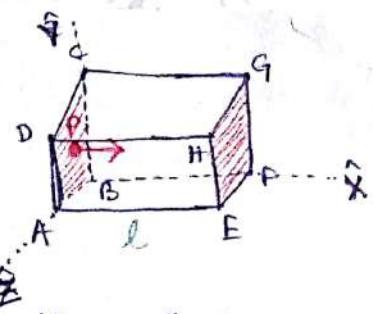
from K.T. (a) collisionless atoms in a box moving in 3 directions, (b) collisionless atoms coming from all directions.

collision will be dealt in "mean free path".

Method 1

$$AB = AD = AE = l$$

The gas is confined within this cube of volume l^3 . P (say) is a gas atom with velocity "c" whose components in 3-direction is (u, v, w) .



N = total no. of atoms or molecules.

So each of them have different velocity $c_1, c_2, c_3, c_4, \dots$ etc
 so different components $(u_1, v_1, w_1), (u_2, v_2, w_2), (u_3, v_3, w_3), \dots$

$$\begin{aligned}
 \text{Mean square average } \bar{c^2} &= \frac{c_1^2 + c_2^2 + c_3^2 + \dots}{N} = \frac{u_1^2 + u_2^2 + u_3^2 + \dots}{N} \\
 &\quad + \frac{v_1^2 + v_2^2 + v_3^2 + \dots}{N} + \frac{w_1^2 + w_2^2 + w_3^2 + \dots}{N} \\
 &= \bar{u^2} + \bar{v^2} + \bar{w^2} \quad - \textcircled{1}
 \end{aligned}$$

mean square velocity in x-direction ↓ y-direction → t-direction.

Consider particle p with mass m_p , velocity $\vec{c} = (\bar{u}, \bar{v}, \bar{w})$.

It travels from ABCD to EFGH, makes collision to exert pressure, rebounds elastically, momentum gets changed, comes back to ABCD to make another collision.

Total distance traveled with velocity u is $2l$.

\therefore Time between collision = $\frac{2l}{u}$, meaning
number of collision per second = $\frac{u}{2l}$.

Momentum imparted in $+x$ direction of on EFGH = mu .
 Momentum obtained in $-x$ direction after collision = $-mu$

$$\therefore \text{Change of momentum} = mu - (-mu) = 2mu.$$

Rate of change of momentum for one atom in x direction

$$= 2mu \times \frac{u}{2l} = \frac{mu^2}{l}$$

Similarly in Y and Z direction, rate of change of momentum is $\frac{mv^2}{l}$ & $\frac{mw^2}{l}$ for one atom

∴ Total rate of change of momentum for all atoms per unit area along X direction is

$$P_x = \frac{m(u_1^2 + u_2^2 + u_3^2 + \dots)}{l} \times \frac{1}{l^2} = m\bar{u}^2 \frac{N}{l^3} = mn\bar{u}^2 \quad (\text{see eq. } ①)$$

Similarly $P_y = mn\bar{v}^2$, $P_z = mn\bar{w}^2$.

In steady state, molecules move in all directions, so no preference, meaning $\bar{u}^2 = \bar{v}^2 = \bar{w}^2$ & $P_x = P_y = P_z$.

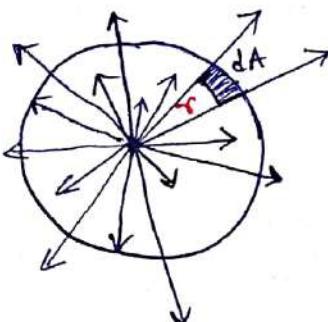
Meaning $\bar{u}^2 = \bar{v}^2 = \bar{w}^2 = \frac{1}{3}\bar{c}^2$ (see eq. ①)

Collecting all pieces together,

$$P_x = P_y = P_z = \frac{1}{3}mn\bar{c}^2 \quad \text{or}$$

$$P = \frac{1}{3}mn\bar{c}^2$$

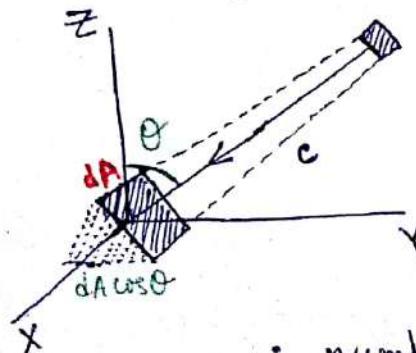
Method 2



N no. of molecules moving in all directions with all possible velocity. How many collide with vessel & insert pressure?

number of vectors per unit area = $\frac{N}{4\pi r^2}$

∴ number of molecules at dA is $\frac{N dA}{4\pi r^2}$



We already learned that $dA = r^2 \sin\theta d\theta d\phi$

$$\frac{N dA}{4\pi r^2} = \frac{N}{4\pi} \sin\theta d\theta d\phi$$

∴ number of molecules per unit volume within velocity range $c \text{ to } c+dc$ [dn_c], within direction $\theta \text{ to } \theta+d\theta$ & $\phi \text{ to } \phi+d\phi$ [$dw = \sin\theta d\theta d\phi$]

$$dn_{c,\theta,\phi} = \frac{dn_c}{4\pi} \sin\theta d\theta d\phi$$

Let's find now, how many of them strike dA of the wall of container. Geometrically, this is the number of molecules within the slanted prism of length $c dt$ with edges in the direction θ & $\phi = \frac{dn_c}{4\pi} \sin \theta d\theta d\phi \times cdA \cos \theta dt$

\therefore Total number of collisions at dA per unit time

$$\int_{c=0}^{\infty} \int_{\theta=0}^{\pi/2} \int_{\phi=0}^{2\pi} \frac{dn_c}{4\pi} \sin \theta d\theta d\phi \times cdA \cos \theta$$

$$= \frac{dA}{4\pi} \int_0^{\infty} c dn_c \int_0^{\pi/2} \sin \theta \cos \theta d\theta \int_0^{2\pi} d\phi = \frac{dA}{4} \int_0^{\alpha} c dn_c.$$

If n_i atoms per unit volume moves with velocity c_1 ,

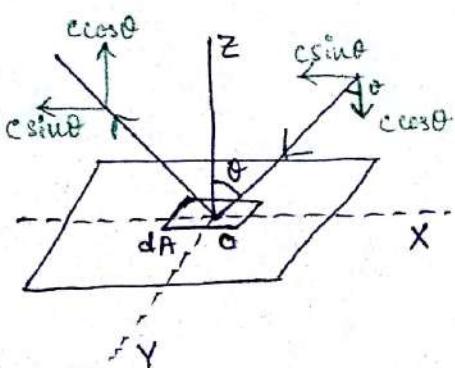
n_2 " " " " " " " " c_2 ,

n_3 " " " " " " " " c_3, \dots

average velocity $\bar{c} = \frac{n_1 c_1 + n_2 c_2 + n_3 c_3 + \dots}{n_1 + n_2 + n_3 + \dots}$

$$= \frac{\sum n_i c_i}{\sum n_i} = \frac{1}{n} \int_0^{\infty} c dn_c$$

\therefore Number of molecules colliding at dA of the container per unit time = $\boxed{\frac{dA}{4} n \bar{c}}$



Now let's compute change in momentum by molecules striking area dA in unit time.

normal component of incident momentum is $mc \cos \theta$ & reflected momentum $-mc \cos \theta$.

So each atom had a change in momentum $2mc \cos\theta$.

∴ Total change of momentum experienced by all gas atoms/molecules colliding to area dA , per unit time

$$\therefore \int_{c=0}^{\infty} \int_{\theta=0}^{\pi/2} \int_{\phi=0}^{2\pi} \frac{dn c}{4\pi} \sin\theta d\theta d\phi \times c dA \cos\theta \times 2mc \cos\theta$$

$$= \frac{mdA}{2\pi} \int_{c=0}^{\infty} c^2 dn c \int_{\theta=0}^{\pi/2} \cos^2\theta \sin\theta d\theta \int_{\phi=0}^{2\pi} d\phi = \frac{1}{3} mdA \int_{c=0}^{\infty} c^2 dn c$$

$$C_{rms}^2 = \bar{c}^2 = \frac{n_1 c_1^2 + n_2 c_2^2 + n_3 c_3^2 + \dots}{n_1 + n_2 + n_3 + \dots} = \frac{\sum n_i c_i^2}{\sum n_i}$$
$$= \int_{0}^{\infty} c^2 dn c$$

∴ Force exerted by gas atoms on dA is

$$F = \frac{1}{3} mdA n \bar{c}^2$$

Thus, pressure exerted

$$P = \frac{F}{dA} = \frac{1}{3} mn \bar{c}^2$$

Corollary from above, $P = \frac{1}{3} \rho \bar{c}^2 \Rightarrow \bar{c} = \sqrt{\frac{3P}{\rho}}$

for Hydrogen $\rho = 8.9 \times 10^{-5}$ gm/cc.

1 atm pressure $P = h\rho g = 76 \times 13.6 \times 981$ dynes/cm²

$$\therefore \bar{c} = \sqrt{\frac{3 \times 76 \times 13.6 \times 981}{8.9 \times 10^{-5}}} = 1.85 \times 10^5 \text{ cm/sec.}$$

$$c_l = 3 \times 10^8 \text{ m/s}, c_s = 300 \text{ m/s}$$
$$= 3 \times 10^10 \text{ cm/s}, c = 3 \times 10^4 \text{ cm/s.}$$

Kinetic interpretation of temperature

From K.T. $p = \frac{1}{3} m \bar{v}^2 = \frac{1}{3} m \frac{N}{V} \bar{c}^2$

But we assume!!

$$\therefore PV = \frac{1}{3} m N \bar{c}^2 = RT \quad [\text{Boyle's law}]$$

$$\bar{c} = \sqrt{\frac{3RT}{mN}} = \sqrt{\frac{3RT}{M}} \quad \text{where } M = \text{molecular weight}$$

$\therefore \bar{c} \propto \sqrt{T}$ RMS velocity of gas atom is proportional to square root of absolute temperature.

As from $T=0$, $\bar{c}=0$ i.e. absolute zero temperature is where molecule cease to move.

$$\text{Now } \bar{c}^2 = \frac{3RT}{M} \Rightarrow \frac{1}{2} M \bar{c}^2 = \frac{3}{2} RT$$

divide by N , $\frac{1}{2} m \bar{c}^2 = \frac{3}{2} k_B T$, $k_B = \text{Boltzmann's constant.}$

mean K.E. ↓
 abs. temp.

for a given T , there is always a K.E. & molecular collision lead to uniform T .

Boyle's law from K.T.

$$PV = \frac{1}{3} M \bar{c}^2 \text{ & because } \bar{c}^2 \propto T$$

so if T is fixed \bar{c}^2 is constant so $PV = \text{constant.}$

Charles's law from K.T.

Again $\bar{c}^2 \propto T$, so $PV \propto T$.

i.e. $V \propto T$ when $P = \text{constant.}$

Avogadro's Law from K.T.

For same T & P, equal V of gases contain equal number of atoms.

$$\begin{array}{c} \boxed{P, V,} \\ 1 \\ N_1 \end{array} \quad \begin{array}{c} \boxed{P, V,} \\ 2 \\ N_2 \end{array}$$

$$P = \frac{1}{3} m_1 \frac{N_1}{V} \bar{c}_1^2 \quad \therefore m_1 N_1 \bar{c}_1^2 = m_2 N_2 \bar{c}_2^2$$

$$P = \frac{1}{3} m_2 \frac{N_2}{V} \bar{c}_2^2$$

But T is equal, so K.E. is equal.

$$\frac{1}{2} m_1 \bar{c}^2 = \frac{1}{2} m_2 \bar{c}_2^2 \Rightarrow m_1 \bar{c}_1^2 = m_2 \bar{c}_2^2$$

$$\therefore N_1 = N_2$$

Clapeyron's equation from K.T.

$$P = \frac{1}{3} m n \bar{c}^2 = \frac{1}{3} \frac{n}{N} m N \bar{c}^2 = \frac{n}{N} \times \frac{1}{3} M \bar{c}^2$$

$$= \frac{n}{N} RT = n k_B T \quad [k_B = \frac{R}{N}]$$

$$N = 6.023 \times 10^{23} \text{ atoms/mole.}$$

Universal gas constant R

$$PV = RT$$

$$\therefore R = \frac{PV}{T} = \frac{(76 \times 13.6 \times 981) \times 22.4 \times 10^3}{273}$$

$$= 8.31 \times 10^7 \text{ dynes-cm/}^\circ\text{K/mole or}$$

$$\text{erg/deg/mole.}$$

$$\text{in heat units, } = \frac{8.31 \times 10^7}{4.18 \times 10^7} = 2 \text{ cal/degK/mole}$$

$$\& k_B = \frac{R}{N} = \frac{8.31 \times 10^7}{6.023 \times 10^{23}} = 1.38 \times 10^{-16} \text{ ergs/degK/mole}$$

Dalton's law of partial pressure

N no. of gases with density ρ_1, ρ_2, ρ_3 with rms velocities $\bar{c}_1, \bar{c}_2, \bar{c}_3$ etc., $P = \frac{1}{3} \rho_1 \bar{c}_1^2 + \frac{1}{3} \rho_2 \bar{c}_2^2 + \frac{1}{3} \rho_3 \bar{c}_3^2 = p_1 + p_2 + p_3 + \dots$

$$\bar{c}_1, \bar{c}_2, \bar{c}_3 \text{ etc., } P = \frac{1}{3} \rho_1 \bar{c}_1^2 + \frac{1}{3} \rho_2 \bar{c}_2^2 + \frac{1}{3} \rho_3 \bar{c}_3^2 = p_1 + p_2 + p_3 + \dots$$

Relation between pressure & K.E.

$$P = \frac{1}{3} \rho \bar{c}^2 = \frac{2}{3} \frac{1}{2} \rho \bar{c}^2 = \frac{2}{3} E$$

so pressure of a gas is $\frac{2}{3}$ the translational K.E. of the atoms per unit volume.

CW Compute the r.m.s. velocity of oxygen & $\overset{(O_2)}{CO_2}$ hydrogen atoms at $27^\circ C$. Given, density of oxygen at N.T.P. = 1.43 kg/m^3 & molecular weight of CO_2 = 44 gm , (molecular weight of O_2 = 32 gm).

at N.T.P. ($T_0 = 273K$) for O_2 , $\rho_0 = 1.43 \text{ kg/m}^3$.

[If not this supplied, then molecular weight = 32 gm

$$= 0.032 \text{ kg}$$

at N.T.P. gram molecular volume = 22.4 litre
 $= 22.4 \times 10^{-3} \text{ m}^3$
 $= 0.0224 \text{ m}^3$.

$$\text{Density } \rho_0 = \frac{0.032}{0.0224} = 1.43 \text{ kg/m}^3]$$

Similarly for CO_2 , density $\rho_0 = \frac{0.044}{0.0224} = 1.96 \text{ kg/m}^3$

at N.T.P.

Using K.T. we have $\rho_0 T_0 = \rho T$.

$$\therefore \rho_{27^\circ} = \frac{\rho_0 T_0}{T} = \frac{1.43 \times 273}{(27+273)} = 1.3 \text{ kg/m}^3 \text{ for } O_2$$

$$\rho_{27^\circ} = \frac{\rho_0 T_0}{T} = \frac{1.96 \times 273}{(27+273)} = 1.79 \text{ kg/m}^3 \text{ for } CO_2$$

$$\text{pressure } P = 0.76 \times 13.6 \times 10^3 \times 9.8 = 1.013 \times 10^5 \text{ N/m}^2$$

from K.T. $P = \frac{1}{2} \rho \bar{C}^2$

So r.m.s. velocity $\bar{C}_{O_2} = \sqrt{\frac{3P}{\rho_{O_2}}} = \sqrt{\frac{3 \times 1.013 \times 10^5}{1.3}} = 4.835 \times 10^2 \text{ m/s.}$

$$\bar{C}_{CO_2} = \sqrt{\frac{3P}{\rho_{CO_2}}} = \sqrt{\frac{3 \times 1.013 \times 10^5}{1.79}} = 4.12 \times 10^2 \text{ m/s.}$$

Ques. 1. Calculate the number of molecules/cc of an ideal gas at $27^\circ C$ & at pressure of 20 mm of mercury
Density of mercury = 13.6 gm/cc & mean KE of a molecule at $27^\circ C$ is 4×10^{-21} joules.

2. At what temperature will the r.m.s. velocity of a gas will become half its value at $0^\circ C$?

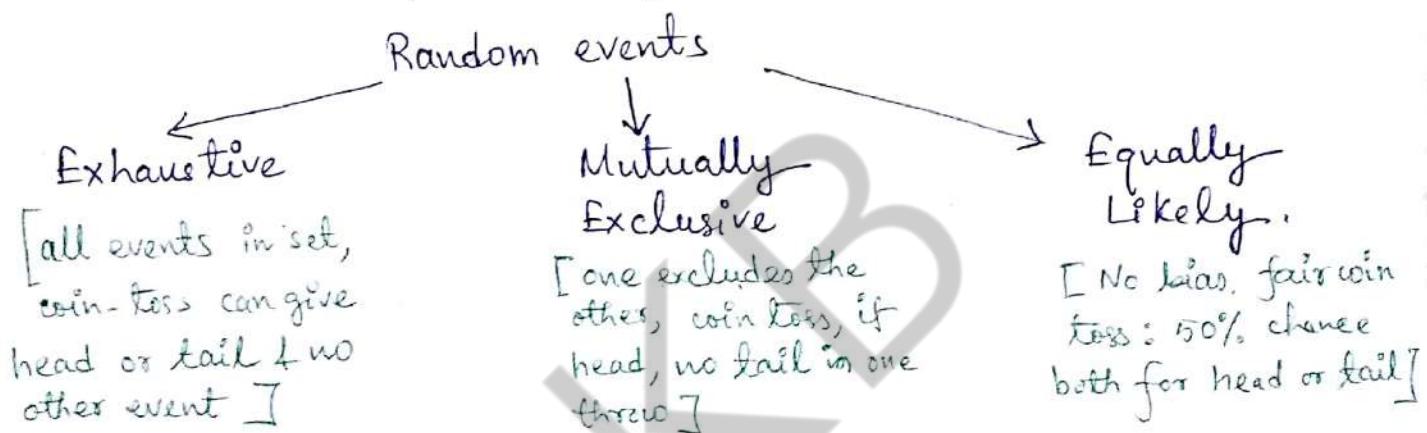
Maxwell-Boltzmann law of distribution of velocity

The question is what is dnc ? Physically dnc is no. of atoms per unit volume within velocity c & $c+dc$.

Can we calculate dnc ? $dnc \stackrel{?}{=} f(P, T)$.

J.C. Maxwell computed it in 1859.

Let's digress & an excursion to random events & what we mean by "probability".



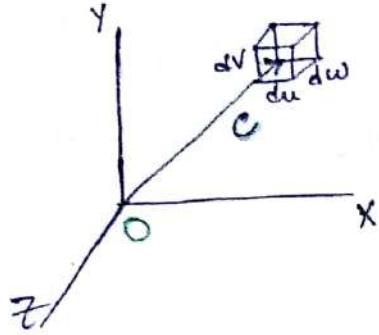
If there are N number of exhaustive, mutually exclusive and equally likely events of which M number are favourable to event A , then

$$P(A) = \frac{M}{N}$$

If two events A & B mutually exclusive, then total probability of either of them to happen in a trial is $P(A) + P(B)$

If two events A & B happen independently, then total probability of both events happening simultaneously in a trial is $P(A)P(B)$.

If x is random variable defined by a function $f(x)$,
then $f(x)dx = \text{probability of a variate falling within}$
 $x \& x+dx.$



- Assumptions:
- (a) density is uniform & velocity in all direction is equal.
 - (b) isotropy \rightarrow results independent of coordinate system.
 - (c) velocities in any 3 coordinates is independent

If a molecule at O has velocity $\vec{c} = (u\hat{i}, v\hat{j}, w\hat{k})$
then $c^2 = u^2 + v^2 + w^2$. Components u, v, w can change as
 \vec{c} changes direction but magnitude of $c = \text{constant}$.

$$\therefore dc^2 = 0 = 2udu + 2vdv + 2wdw$$

$$\text{So } udu + vdv + wdw = 0 \quad \text{--- (1)}$$

This means du, dv & dw are not independent.

Probability that an atom has x component of velocity
 u & $u+du$ is $f(u)du$, mathematically, $P_u = \frac{dn_u}{n} = f(u)du$.
 n = number density.

Similarly, between v & $v+dv$ is $P_v = \frac{dn_v}{n} = f(v)dv$

" " w & $w+dw$ is $P_w = \frac{dn_w}{n} = f(w)dw$.

As they're independent*, the total probability is

$$P_{u,v,w} = \frac{dn_{u,v,w}}{n} = f(u)f(v)f(w) du dv dw$$

$dn_{u,v,w} = n f(u)f(v)f(w) du dv dw$, also means

$$dN_{u,v,w} = N f(u)f(v)f(w) du dv dw$$

So in N number of molecules, $dN_{u,v,w}$ means this many of them are between $u \& u+du$, $v \& v+dv$, $w \& w+dw$.

$$\therefore \text{Molecular density } \rho = \frac{dN_{u,v,w}}{du dv dw} = N f(u) f(v) f(w)$$

& as this is uniform, $d\rho = 0$

$$= f'(u) f(v) f(w) du + f(u) f'(v) f(w) dv + f(u) f(v) f'(w) dw$$

$$\text{Divide by } \frac{1}{f(u) f(v) f(w)} \Rightarrow \frac{f'(u)}{f(u)} du + \frac{f'(v)}{f(v)} dv + \frac{f'(w)}{f(w)} dw = 0 \quad \text{--- (2)}$$

when ① & ② both are true, we invoke Lagrange's undetermined multiplier λ do ① $\times \lambda +$ ②,

$$\left[\frac{f'(u)}{f(u)} + \alpha u \right] du + \left[\frac{f'(v)}{f(v)} + \alpha v \right] dv + \left[\frac{f'(w)}{f(w)} + \alpha w \right] dw = 0$$

If we say, du is dependent, then we choose λ such that

$$\frac{f'(u)}{f(u)} + \alpha u = 0$$

& because dv & dw is dependent, so

$$\frac{f'(v)}{f(v)} + \alpha v = 0, \quad \frac{f'(w)}{f(w)} + \alpha w = 0.$$

$$\therefore \frac{df(u)}{f(u)} = -\alpha u du.$$

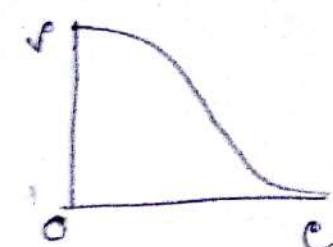
$$\text{Integrating, } \ln f(u) = -\frac{\alpha}{2} u^2 + \ln A$$

$$\text{or } f(u) = A e^{-\frac{\alpha u^2}{2}} = A e^{-\frac{b u^2}{2}} \quad \left\{ b = \frac{\alpha}{2} \right\}$$

$$\text{Similarly, } f(v) = A e^{-\frac{b v^2}{2}}, \quad f(w) = A e^{-\frac{b w^2}{2}}$$

$$\text{So } f = N A^3 e^{-b(u^2+v^2+w^2)} = N A^3 e^{-bc^2}$$

$$dN_{u,v,w} = N A^3 e^{-b(u^2+v^2+w^2)} du dv dw$$



What is remaining now is to find out

constants A & b.

$$\iiint_{-\infty}^{\infty} dN_{u,v,w} = N$$

$$\text{or } NA^3 \int_{-\infty}^{\infty} e^{-bu^2} du \int_{-\infty}^{\infty} e^{-bv^2} dv \int_{-\infty}^{\infty} e^{-bw^2} dw = N$$

$$\left[\text{Now } \int_{-\infty}^{\infty} e^{-bu^2} du \right]$$

$$= \int_{-\infty}^{\infty} e^{-z} \frac{1}{2\sqrt{b}} z^{-\gamma_2} dz$$

$$= \frac{1}{2\sqrt{b}} \int_0^{\infty} e^{-z} z^{-\gamma_2} dz = \frac{\Gamma(\gamma_2)}{\sqrt{b}} = \sqrt{\frac{\pi}{b}}$$

$$A^3 \left(\frac{\pi}{b} \right)^{\gamma_2/2} = 1 \quad \Rightarrow \quad A = \sqrt{\frac{b}{\pi}}$$

$$\text{Let } bu^2 = z$$

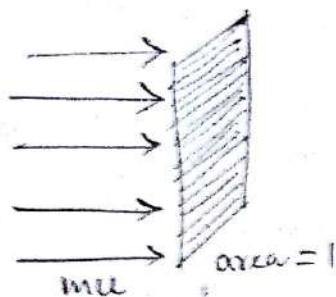
$$2bu du = dz$$

$$\Rightarrow du = \frac{dz}{2b\sqrt{z}}$$

Evaluate b collisions per second

= area \times velocity \times number density at that

$$= 1 \times u \times n_u$$



Change in momentum = $2mu$.

So pressure = rate of change of momentum per unit area.

$$P_u = \sum_{u=0}^{\infty} u n_u \times 2mu = 2m \sum_{u=0}^{\infty} n_u u^2 = 2m \int_0^{\infty} n_u u^2 f(u) du$$

$$= 2m n_u \int_0^{\infty} A e^{-bu^2} u^2 du$$

$$\therefore P_u = 2mn_u A \int_0^\infty e^{-\frac{z^2}{b}} \frac{z}{b} \frac{dz \sqrt{b}}{2b \sqrt{\pi}}$$

[put $b u^2 = z$]

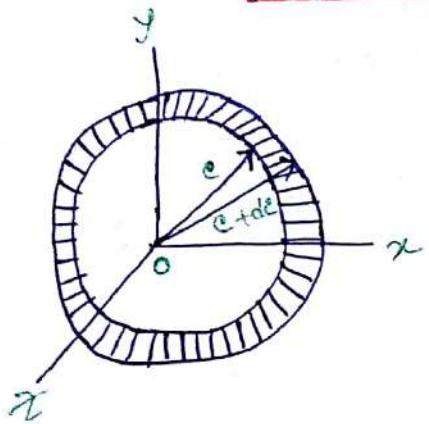
$$= \frac{mn_u A}{2b^{3/2}} \Gamma(\frac{1}{2})$$

$$= \frac{mn_u}{2b^{3/2}} \frac{b^{1/2}}{\pi^{1/2}} \cdot \pi^{1/2} = \frac{mn_u}{2b} = n_u k_B T.$$

[from Clapeyron's equation]

$$\therefore b = \frac{m}{2k_B T}, \quad A = \sqrt{\frac{b}{\pi}} = \sqrt{\frac{m}{2\pi k_B T}}$$

$$\therefore dN_{u,v,w} = N \left(\frac{m}{2\pi k_B T} \right)^{3/2} e^{-\frac{m}{2k_B T}(u^2+v^2+w^2)} du dw dv.$$

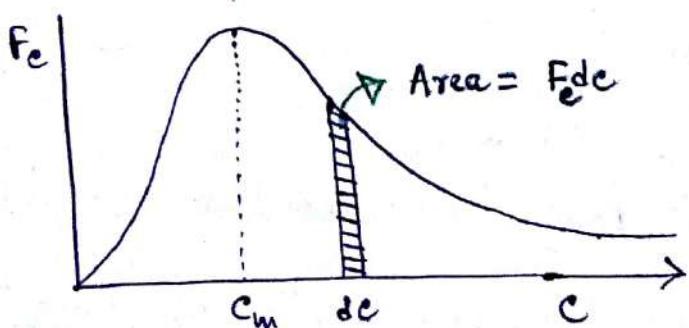


volume between c & $c+dc$ is

$$\begin{aligned} & \frac{4}{3}\pi(c+dc)^3 - \frac{4}{3}\pi c^3 \\ &= \frac{4}{3}\pi c^3 + \frac{4}{3}\pi 3c^2 dc + \frac{4}{3}\pi 3c(dc) + \frac{4}{3}\pi (dc)^3 \\ &\quad - \frac{4}{3}\pi c^3 \\ &= 4\pi c^2 dc. \end{aligned}$$

$$\therefore dN_c = N \left(\frac{m}{2\pi k_B T} \right)^{3/2} e^{-\frac{mc^2}{2k_B T}} 4\pi c^2 dc$$

$$\therefore P_c = \frac{dN_c}{N} = 4\pi \left(\frac{m}{2\pi k_B T} \right)^{3/2} c^2 e^{-\frac{mc^2}{2k_B T}} dc = F_c dc.$$



Remarks : (a) $c \rightarrow 0, F_c \rightarrow 0$

(b) $c \rightarrow \infty, F_c \rightarrow 0$

$$c_m = F_c^{\text{maximum}}$$

= most probable velocity

= maximum no. of atoms
will possess this velocity

Velocity component distribution

What is the number of molecules within velocity u & $u+du$ but any value in \hat{y} or \hat{z} direction.

$$dN_{u,v,w} = N \left(\frac{m}{2\pi k_B T} \right)^{3/2} e^{-\frac{m}{2k_B T}(u^2+v^2+w^2)} du dv dw.$$

$$\begin{aligned} \therefore dN_u &= \int_{-\infty}^{\infty} \int_{-\infty}^{\infty} dN_{u,v,w} \\ &= N \left(\frac{m}{2\pi k_B T} \right)^{3/2} du \int_{-\infty}^{\infty} e^{-\frac{mv^2}{2k_B T}} dv \int_{-\infty}^{\infty} e^{-\frac{mw^2}{2k_B T}} dw \times e^{-\frac{mu^2}{2k_B T}} \end{aligned}$$

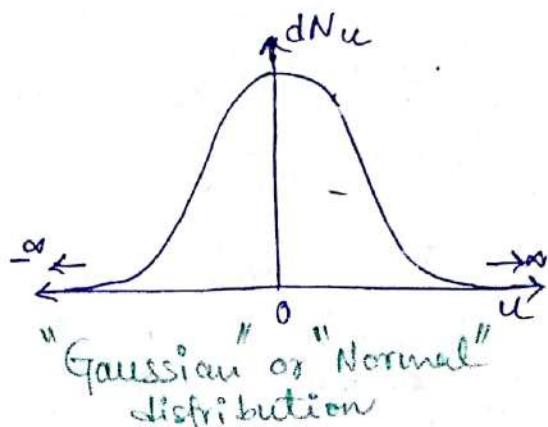
$$\begin{aligned} \left[\text{Now } \int_{-\infty}^{\infty} e^{-\frac{mv^2}{2k_B T}} dv \right] &= 2 \int_0^{\infty} e^{-\frac{mv^2}{2k_B T}} dv \\ &= \frac{2\sqrt{k_B T}}{\sqrt{2\pi m}} \int_0^{\infty} e^{-z^2} z^{-1/2} dz \\ &= \sqrt{\frac{2k_B T}{m}} \times \sqrt{\pi} = \sqrt{\frac{2\pi k_B T}{m}} \end{aligned}$$

$$\therefore dN_u = N \left(\frac{m}{2\pi k_B T} \right)^{3/2} \left(\frac{2\pi k_B T}{m} \right)^{1/2} e^{-\frac{mu^2}{2k_B T}} du$$

$$dN_u = N \left(\frac{m}{2\pi k_B T} \right)^{1/2} e^{-\frac{mu^2}{2k_B T}} du$$

$$\text{Similarly, } dN_v = N \left(\frac{m}{2\pi k_B T} \right)^{1/2} e^{-\frac{mv^2}{2k_B T}} dv$$

$$dN_w = N \left(\frac{m}{2\pi k_B T} \right)^{1/2} e^{-\frac{mw^2}{2k_B T}} dw$$



Average velocity, RMS velocity, Most probable velocity

$$\text{Avg. velocity } \langle c \rangle = \frac{N_1 c_1 + N_2 c_2 + \dots}{N_1 + N_2 + \dots} = \frac{\sum N_i c_i}{\sum N_i}$$

$$= \int_0^\infty c dN_c / N$$

$$= 4\pi \left(\frac{m}{2\pi k_B T} \right)^{3/2} \int_0^\infty c \cdot c^2 e^{-\frac{mc^2}{2k_B T}} dc$$

$$= 4\pi A^3 \int_0^\infty c^3 e^{-bc^2} dc$$

remember,
 $A = \left(\frac{m}{2\pi k_B T} \right)^{1/2}$

$$b = \frac{m}{2k_B T}$$

$$= 4\pi A^3 \int \frac{z}{b} e^{-z^2} \frac{dz}{2b} = \frac{4\pi A^3}{2b^2} \int_0^\infty e^{-z^2} dz$$

$$= \frac{4\pi A^3}{2b^2} \Gamma(2) = \frac{4\pi A^3}{2b^2} = 4\pi \frac{m}{2\pi k_B T} \left(\frac{m}{2\pi k_B T} \right)^{1/2} \times \frac{4k_B T}{2m^2}$$

$$= \left(\frac{8k_B T}{m\pi} \right)^{1/2}$$

put $be^2 = z$
 $2bcdc = dz$

$$\Gamma(2) = 1$$

$$\therefore \boxed{\langle c \rangle = \sqrt{\frac{8k_B T}{m\pi}}}$$

$$\begin{aligned}
 \text{RMS velocity } C_{\text{rms}}^2 &= \frac{\sum N_i c_i^2}{\sum N_i} = \frac{1}{N} \int_0^\infty c^2 dN_c \\
 &= 4\pi A \int_0^\infty c^4 e^{-bc^2} dc \\
 &= 4\pi A \int_0^\infty \frac{z^2}{b^2} e^{-z^2} \frac{dz \sqrt{b}}{2b\sqrt{z}} \\
 &= \frac{4\pi A^3}{2b^{5/2}} \int_0^\infty e^{-z^2} z^{3/2} dz = \frac{4\pi A^3}{2b^{5/2}} \Gamma(5/2) \\
 &= \frac{A\pi A^3}{2b^{5/2}} \frac{3}{2} \times \frac{1}{2} \times \sqrt{\pi} = \frac{3\pi^{3/2}}{2} \left(\frac{m}{2\pi k_B T}\right)^{3/2} \left(\frac{2k_B T}{m}\right)^{5/2} \\
 &= \frac{3k_B T}{m}
 \end{aligned}$$

put, $bc^2 = z$
 $2bc dc = dz$
 $dc = \frac{dz}{2b\sqrt{z}}$

$$\therefore C_{\text{rms}} = \sqrt{\frac{3k_B T}{m}}$$

Most probable velocity C_m is $\frac{dF_c}{dc} \Big|_{c=c_m} = 0$.

$$\begin{aligned}
 \therefore \frac{d}{dc} \left\{ 4\pi A c^3 e^{-bc^2} \right\} \Big|_{c=c_m} &= 0 \\
 \text{or } \frac{d}{dc} \left\{ c^2 e^{-bc^2} \right\} \Big|_{c=c_m} &= 0 \\
 \text{or } \left\{ 2ce^{-bc^2} - c^2 2bc e^{-bc^2} \right\} \Big|_{c=c_m} &= 0 \\
 \text{or } \left\{ 2ce^{-bc^2} (1 - bc^2) \right\} \Big|_{c=c_m} &= 0.
 \end{aligned}$$

This can be true if $c \rightarrow \infty$ (unphysical) or $\{1 - bc^2\}_{c=c_m} = 0$

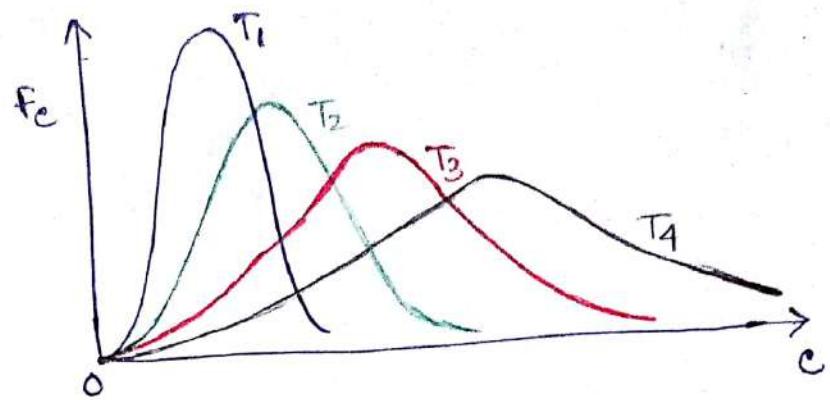
$$\therefore bC_m^2 = 1 \quad \text{or} \quad C_m = \frac{1}{\sqrt{b}} = \sqrt{\frac{2k_B T}{m}}$$

or $C_m = \sqrt{\frac{2k_B T}{m}}$

Note $C_{\text{rms}} > \langle c \rangle > C_m$

Corollary $\int_0^\infty F_c \, dc = 1$

$$T_4 > T_3 > T_2 > T_1$$



Also, no. of molecules colliding per unit area per unit time

$$dn = \frac{1}{4} n \bar{c} = \frac{1}{4} n \sqrt{\frac{8k_B T}{m\pi}} = \frac{1}{4} \frac{P}{k_B T} \sqrt{\frac{8k_B T}{m\pi}} \quad (\text{as } P = nk_B T)$$

$$dn = \frac{P}{\sqrt{2m\pi k_B T}}$$

CW 1. The velocity distribution in two dimension is

$$dn_{u,v} = n \left(\frac{m}{2\pi k_B T} \right) e^{-\frac{m(u^2+v^2)}{2k_B T}} \, du \, dv.$$

find the distribution of molecular speed. Using that, find

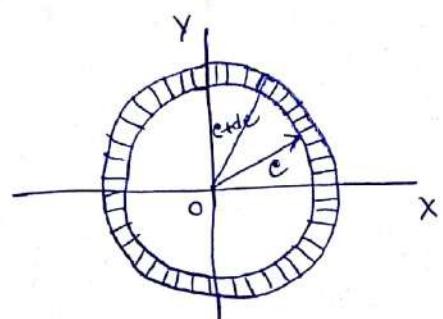
$$c_m, \bar{c}, c_{rms}^2.$$

$$\bar{c}^2 = u^2 + v^2.$$

Take two concentric circles between velocity c & $c+dc$, area

$$dudv = \pi(c+dc)^2 - \pi c^2 = 2\pi c dc.$$

$$\therefore dn_c = n \left(\frac{m}{2\pi k_B T} \right) e^{-\frac{mc^2}{2k_B T}} 2\pi c dc = f_c dc.$$

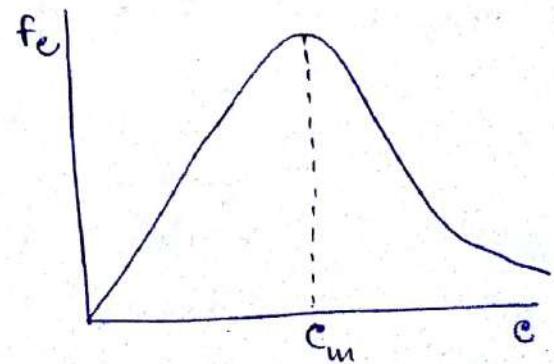


$$\frac{df_c}{dc} \Big|_{c=c_m} = 0$$

$$\therefore \frac{d}{dc} (ce^{-\frac{mc^2}{2k_B T}}) = 0$$

$$\text{or } 1 - c_m^2 \frac{m}{k_B T} = 0 \quad \text{or } c_m = \sqrt{\frac{k_B T}{m}}$$

please also calculate $\frac{1}{n} \int_0^\infty c \, dn_c$ & $\frac{1}{n} \int_0^\infty c^2 \, dn_c$.



convince yourself that $c_{rms} = \sqrt{\frac{2k_B T}{m}}$ and $\bar{c} = \sqrt{\frac{\pi k_B T}{2m}}$.

2. Using Maxwell velocity distribution, calculate the probability that the velocity of O_2 molecule lies between 100 m/s & 101 m/s at $-73^\circ C$.

$$dN_c = 4\pi N \left(\frac{m}{2\pi k_B T} \right)^{3/2} e^{-\frac{mc^2}{2k_B T}} c^2 dc.$$

$$\therefore \text{Probability } P = \frac{dN_c}{N} = 4\pi \left(\frac{m}{2\pi k_B T} \right)^{3/2} e^{-\frac{mc^2}{2k_B T}} c^2 dc. \quad \text{--- (1)}$$

$$\text{Now } m = \frac{M}{N} = \frac{32 \text{ gm}}{6.023 \times 10^{23}} = 5.31 \times 10^{-26} \text{ kg.}$$

$$T = -73^\circ C = 200 \text{ K.}, \quad c = 100 \text{ m/s}, \quad dc = 101 - 100 = 1 \text{ m/s.}$$

$$\begin{aligned} \therefore P &= 4\pi \left[\frac{5.31 \times 10^{-26}}{2\pi \times 1.38 \times 10^{-23} \times 200} \right]^{3/2} \times \exp \left[-\frac{5.31 \times 10^{-26} \times 10^4}{2 \times 1.38 \times 10^{-23} \times 200} \right] \times 10^4 \times 1 \\ &= 4\pi \times 5.36 \times 10^{-9} \times 0.9 \times 10^4 = 6.06 \times 10^{-4} = 0.06\% \end{aligned}$$

3. Compute the fraction of molecules of a gas possessing speeds within 1% of the most probable speed.

$$c_m = \sqrt{\frac{2k_B T}{m}}$$

fraction = probability P in equation (1) above. with $c=c_m$

$$P = 4\pi \left(\frac{m}{2\pi k_B T} \right)^{3/2} e^{-\frac{m}{2k_B T} \frac{2k_B T}{m}} \frac{2k_B T}{m} dc_m$$

~~dc_m is $\pm 1\%$ of c_m~~ As $2c_m$

As c varies within 1% of $c_m \Rightarrow [0.99c_m, 1.01c_m]$.

$$\therefore dc_m = (1.01 - 0.99)c_m = 0.02 \times \sqrt{\frac{2k_B T}{m}}.$$

$$\begin{aligned} \therefore P &= \frac{1}{\sqrt{\pi}} \left(\frac{m}{2k_B T} \right)^{3/2} e^{-\frac{1}{2k_B T} \frac{2k_B T}{m}} \sqrt{\frac{2k_B T}{m}} \times 0.02 \\ &= 0.016 = 1.6\% . \end{aligned}$$

HW

1. At what value of speed c will the Maxwell's distribution F_c yield same magnitude for a mixture of

hydrogen & helium gases at 27°C ?

2. Find $\langle c \rangle$ using F_c .

3. Molecular mass of an ideal gas of O_2 is 32. Calculate c_m, \bar{c}, c_{rms} of the gas at 27°C . (Given $R = 8.3 \text{ J}/\text{K mol}$)

4. Convince yourself that $\frac{RT}{m} = \frac{P}{\rho}$. Using that, calculate c_m, \bar{c}, c_{rms} of the molecules of gas at density $1.293 \times 10^{-3} \text{ gm/cc}$ at 76 cm of Hg pressure.

5. The quantity $(c - \bar{c})^2 = \bar{c}^2 - 2c\bar{c} + \bar{c}^2$ is squared deviation of atomic speed from average speed. Calculate the average value of this using Maxwell distribution & obtain the rms deviation.

Maxwell's distribution in reduced format

$$dN_c = 4\pi N \left(\frac{m}{2\pi k_B T} \right)^{3/2} e^{-mc^2/2k_B T} c^2 dc$$

with respect to $C_m = \sqrt{\frac{2k_B T}{m}}$, non dimensionalized $U = \frac{c}{C_m}$ velocity

Substitute $c = \sqrt{\frac{2k_B T}{m}} U$,

$$dN_c = 4\pi N \left(\frac{m}{2\pi k_B T} \right)^{3/2} \frac{2k_B T}{m} U^2 \sqrt{\frac{2k_B T}{m}} dU e^{-\frac{m}{2k_B T} \frac{2k_B T}{m} U^2}$$

$$dN_U = \frac{4N}{\sqrt{\pi}} U^2 e^{-U^2} dU$$

This distribution is independent of temperature.

Energy distribution of the gaseous atoms

What we are after is to find the no. of atoms having K.E. between E & $E + dE$. As they don't have P.E.

$$E = \frac{1}{2}mc^2 \quad \text{or} \quad dE = mcdc.$$

$$\text{or} \quad dc = \frac{dE}{m\sqrt{\frac{2E}{m}}} = \frac{dE}{\sqrt{2mE}}.$$

From Maxwell's velocity distribution, we have

$$dn_c = 4\pi n c^2 \left(\frac{m}{2\pi k_B T}\right)^{3/2} e^{-mc^2/2k_B T} dc.$$

$$\therefore dn_E = 4\pi n \frac{2E}{m} \left(\frac{m}{2\pi k_B T}\right)^{3/2} e^{-E/k_B T} \frac{dE}{\sqrt{2mE}}$$

$$dn_E = 2n \sqrt{\frac{E}{\pi}} (k_B T)^{-3/2} e^{-E/k_B T} dE$$

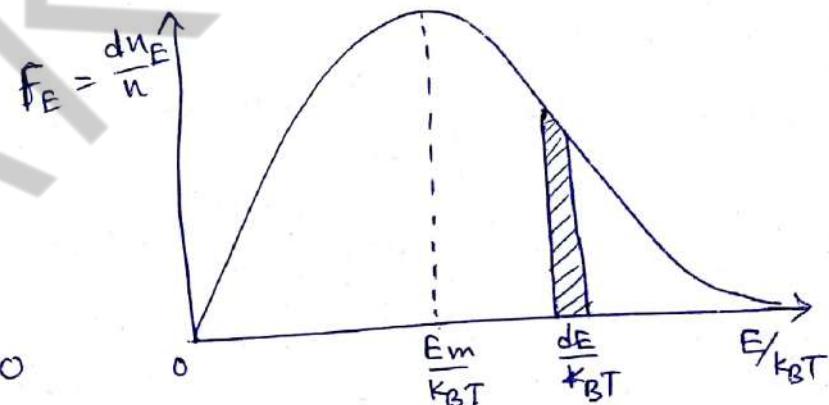
(remember if V is cancelled $dN_E \sim N$)

$$\text{fraction of atoms } \frac{dn_E}{n} \text{ or } \frac{dN_E}{N} = \frac{2}{\sqrt{\pi}} (k_B T)^{-3/2} \sqrt{E} e^{-E/k_B T} dE = f_E dE$$

$$f_E = \frac{2}{\sqrt{\pi}} (k_B T)^{-3/2} \sqrt{E} e^{-E/k_B T}$$

$$\text{at } E = E_m, \frac{df_E}{dE} = 0$$

$$\frac{d}{dE} \left\{ \sqrt{E} e^{-E/k_B T} \right\}_{E=E_m} = 0$$



$$\therefore \frac{1}{2\sqrt{E_m}} e^{-E_m/k_B T} - \frac{\sqrt{E_m}}{k_B T} e^{-E_m/k_B T} = 0$$

$$\therefore \frac{1}{2\sqrt{E_m}} = \frac{\sqrt{E_m}}{k_B T} \quad \text{or} \quad E_m = \frac{k_B T}{2}$$

Compare result with $C_m = \sqrt{\frac{2k_B T}{m}}$. Notice that

$$E_m \neq \frac{1}{2}mc_m^2 = \frac{1}{2}m \frac{2k_B T}{m} = k_B T.$$

Momentum distribution of the gaseous atoms

No. of molecules in a range of momentum p to $p+dp$.

$$p = mc \quad \& \quad dp = mdc.$$

Again, start from Maxwell's distribution

$$dN_c = 4\pi N \left(\frac{m}{2\pi k_B T} \right)^{3/2} c^2 e^{-\frac{mc^2}{2k_B T}} dc$$

$$\therefore dN_p = 4\pi N \left(\frac{m}{2\pi k_B T} \right)^{3/2} \frac{p^2}{m^2} e^{-\frac{p^2}{2k_B T m}} \frac{dp}{m}$$

$$= N \sqrt{\frac{2}{\pi}} (m k_B T)^{-3/2} p^2 e^{-\frac{p^2}{2m k_B T}} dp = f_p dp.$$

at $p = p_m$ (most probable momentum), maximum no. of molecules

lie $\frac{df_p}{dp} \Big|_{p=p_m} = 0 \quad \text{or} \quad \frac{d}{dp} \left(p^2 e^{-\frac{p^2}{2m k_B T}} \right) \Big|_{p=p_m} = 0$

$$\text{or} \quad \frac{d}{dp} (p^2 e^{-bp^2}) \Big|_{p=p_m} = 0$$

$$\text{put } b = \frac{1}{2m k_B T}$$

$$\text{or} \quad (2p e^{-bp^2} - p^2 \cdot 2bp e^{-bp^2}) \Big|_{p=p_m} = 0$$

$$\text{or} \quad [2p e^{-bp^2} (1 - bp^2)] \Big|_{p=p_m} = 0 \quad \Rightarrow \quad p_m = \frac{1}{\sqrt{b}} = \sqrt{2m k_B T}$$

Again, $c_m = \sqrt{\frac{2k_B T}{m}}$

So $mc_m = \sqrt{2m k_B T} = p_m$.

So most of the atoms have velocity c_m are also having the momentum mc_m , but ~~most~~ most of them do not have $\frac{1}{2}mc_m^2$ energy, rather most of the atoms have energy half of that.

If P.E. included in gravitational field $dN_c = 4\pi N \left(\frac{m}{2\pi k_B T} \right)^{3/2} \times e^{-(mgh + \frac{1}{2}mc^2)/k_B T} c^2 dc$

Maxwell-Boltzmann distribution.

Degrees of freedom

1D motion \rightarrow 1 d.o.f.

2D (unconstrained) motion \rightarrow 2 d.o.f.

3D (unconstrained) motion \rightarrow 3 d.o.f.

constraint reduce no. of independent variables necessary to designate the configuration of a dynamical system.

N particles in 3D coordinates $(x_1, y_1, z_1), (x_2, y_2, z_2) \dots (x_N, y_N, z_N)$

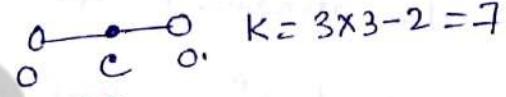
with constraint $F_j(x_1, y_1, z_1, \dots, x_N, y_N, z_N) = 0, j=1, 2, 3, \dots, m.$

$$\therefore K = 3N - m = \text{d.o.f.}$$

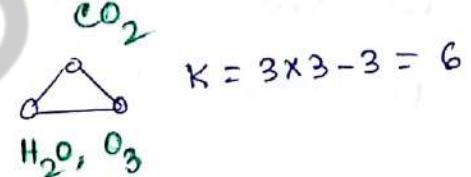
Example (1) Monoatomic atom : $K = 3$ for one atom.

(2) Diatomic atom :  $K = 2 \times 3 - 1 = 5$ for one molecule.

(3) Triatomic molecule : (a) linear



(b) Nonlinear



So trivial case, $m = 3N \rightarrow$ only one configuration.

$m > 3N$ not feasible, more constraint equation than unknowns.

$m < 3N$ is only feasible.

Generalized position & velocity coordinates

If a system has N d.o.f. then we can choose a set of N elements $q_1, q_2, q_3, \dots, q_N$ that correspond to configuration & $\dot{q}_1, \dot{q}_2, \dots, \dot{q}_N$ which to velocity vector at that configuration. These are generalized coordinates.

Law of equipartition of energy

Energy of a dynamical system in thermal equilibrium is equally divided amongst its d.o.f. & the value is $\frac{1}{2}kT$ for each d.o.f.

Suppose a system of free atoms has f d.o.f.

Every d.o.f. gets $\frac{1}{2} k_B T$ energy, then the energy per gm molecule

$$\therefore \Theta = \frac{1}{2} N f k_B T = \frac{1}{2} R f T \quad [k_B = \frac{R}{N}]$$

$$\therefore C_V = \left(\frac{d\Theta}{dT} \right)_V = R f / 2.$$

From $C_P - C_V = R$, we have $C_P = R + \frac{Rf}{2}$.

$$\therefore \frac{C_P}{C_V} = \frac{R + \frac{Rf}{2}}{Rf/2} = \frac{2}{f} + 1 \quad \Rightarrow \quad \gamma = 1 + \frac{2}{f}$$

for monoatomic gas, $K=3$, $\gamma = 1 + \frac{2}{3} = 1.66$.

for diatomic gas, $K=5$, $\gamma = 1 + \frac{2}{5} = 1.4$

for triatomic linear gas, $K=7$, $\gamma = 1 + \frac{2}{7} = 1.28$

for triatomic not collinear gas, $K=6$, $\gamma = 1 + \frac{2}{6} = 1.33$.

Dulong & Petit's law

1819 Experimental investigation \Rightarrow product of atomic weight & specific heat of almost all solid substances are nearly equal to 6 calories.

Consider 1 mole of a solid with each atom/molecule have 3 translational d.o.f. & 2 rotational d.o.f. & 3 vibrational d.o.f. (vibration about mean equilibrium configuration). Rotation isn't allowed in crystal, as they're rigidly fixed.

\therefore Number of d.o.f. of 1 mole solid is $6N$.

Every d.o.f. gets $\frac{1}{2} k_B T$ energy from Equipartition theorem.

$$\therefore \text{Total energy } U = 6N \cdot \frac{1}{2} k_B T = 3N k_B T = 3RT$$

$$\therefore C_V = \left(\frac{dU}{dT} \right)_V = 3R = 3 \times 1.98 \text{ cal} \approx 6 \text{ cal.}$$

Sometimes "atomicity" is also equivalent to d.o.f, $f = \frac{2}{\gamma - 1}$.

Variation of γ with temperature

High T, translation + rotation + vibration,

$$f(\text{diatomic molecule}) = 7, \quad \gamma = 1 + \frac{2}{7} = 1.29.$$

like $H_2 > 5000^\circ C$ $C_V = \frac{7}{2}R$

Medium T, translation + rotation

$$f(\text{diatomic molecule}) = 5, \quad \gamma = 1 + \frac{2}{5} = 1.4.$$

like $H_2 \approx 250K$ $C_V = \frac{5}{2}R$.

Low T, translation

$$f(\text{diatomic molecule}) = 3, \quad \gamma = 1 + \frac{2}{3} = 1.66$$

like $H_2 \approx 70K$ $C_V = \frac{3}{2}R$.

CW 1. What is the atomicity of a gas with heat capacity ratio 1.33? Given, $R = 8.31 \text{ J/mol/K}$. Calculate C_p, C_V values also.

$$C_p - C_V = R \quad \text{or} \quad C_V \left(\frac{C_p}{C_V} - 1 \right) = R \quad \text{or} \quad C_V (\gamma - 1) = R.$$

$$\therefore C_V = \frac{R}{\gamma - 1} = \frac{8.31}{1.33 - 1} = 25.18 \text{ J/mol/K}.$$

$$C_p = \gamma C_V = 33.49 \text{ J/mol/K}.$$

$$\therefore \text{Atomicity } f = \frac{2}{\gamma - 1} = \frac{2}{0.33} = 6. \quad (\text{not monoatomic gas}).$$

2. Calculate (a) K.E. of translation of O_2 molecule, (b) total K.E. and (c) total K.E. of a gram molecule at $27^\circ C$. Given

$$k_B = 1.37 \times 10^{-23} \text{ J/K} \quad N = 6.02 \times 10^{23}$$

$O_2 = 3 \text{ translation} + 2 \text{ rotation } (\theta, \phi), \quad f = 5.$
(x, y, z)

$$(a) \text{ K.E. of translation} = \frac{3}{2} k_B T = \frac{3}{2} \times 1.37 \times 10^{-23} \times 300 \\ = 6.17 \times 10^{-21} \text{ J.}$$

$$(b) \text{ K.E. of total} = \frac{5}{2} k_B T = 10.275 \times 10^{-21} \text{ J}$$

$$(c) \text{ of 1 gram molecule} = \frac{5}{2} k_B T \times N = 10.275 \times 10^{-21} \times 6.02 \times 10^{23} \\ = 6185.55 \text{ J.}$$

γ for a mixture of ideal gases

At temperature T , N_1 moles of ideal gas with f_1 d.o.f.
 N_2 moles of ideal gas with f_2 d.o.f.
 N_3 moles of ideal gas with f_3 d.o.f.
 $\dots \dots \dots \dots \dots$
 N_N moles of ideal gas with f_N d.o.f.

$$\text{Total internal energy } U = \frac{1}{2} k_B T N_1 f_1 N + \frac{1}{2} k_B T N_2 f_2 N + \dots + \frac{1}{2} k_B T N_N f_N N$$

$$\therefore C_V = \left(\frac{dU}{dT} \right)_V = \frac{1}{2} k_B N (N_1 f_1 + N_2 f_2 + \dots + N_N f_N)$$

$$\text{Also } C_P - C_V = (N_1 + N_2 + \dots + N_N) k_B T \quad [R/N = k_B]$$

$$\begin{aligned} \therefore C_P &= (N_1 + N_2 + \dots + N_N) k_B T + \frac{1}{2} (N_1 f_1 + N_2 f_2 + \dots + N_N f_N) k_B N \\ &= \frac{(2+f_1)N_1 + (2+f_2)N_2 + \dots + (2+f_N)N_N}{2} k_B N \end{aligned}$$

$$\therefore \gamma = \frac{C_P}{C_V} = \frac{(2+f_1)N_1 + (2+f_2)N_2 + \dots + (2+f_N)N_N}{f_1 N_1 + f_2 N_2 + \dots + f_N N_N}$$

Example

$$1 \text{ mole of monoatomic gas } (f=3) \text{ & } 1 \text{ mole of diatomic gas } (f=5), \quad \gamma = \frac{(2+f_1)N_1 + (2+f_2)N_2}{f_1 N_1 + f_2 N_2} = \frac{5+7}{3+5} = 1.5$$

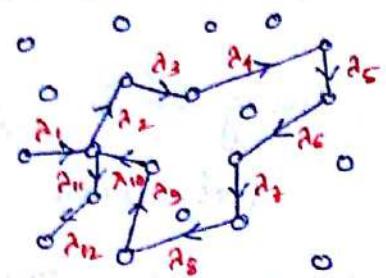
$$1 \leq \gamma \leq 1.67$$

Mean free path We calculated that K.T. gives $\approx 1 \text{ km/s}$ velocity for molecular movement. But we see clouds suspended in air holds together for hours. So there must be some factors that prevent the free escape of atoms.

Clausius showed that such discrepancy goes away if we take small & finite volume for atoms & they change velocity & direction of motion in the process of collision, zigzag path (discrete)

In between two successive collision, the traversed path is free path ($\lambda_1, \lambda_2, \dots, \lambda_n$).

$$\text{Mean free path} = \frac{\lambda_1 + \lambda_2 + \lambda_3 + \dots + \lambda_N}{\text{Number of collision}}$$



Collision probability

Suppose collision rate is P , average velocity of an atom is \bar{c} & in time t , distance covered = $\bar{c}t$ & number of collisions suffered is N . then $N_{\bar{c}} = \frac{\bar{c}t}{P} = \frac{\bar{c}}{P} t$.

Before we calculate "A", let's compute the distribution of λ , meaning probability of an atom moving a distance x without collision, say $f(x)$. This means that $f(x+dx)$ is the probability that atom traverses $x+dx$ length without collision.

If p is collision probability per unit time, then for N atoms number of collisions in time $t = \frac{1}{2} N P t$. ($\frac{1}{2}$ because each collision between 2 atoms is counted twice).

Probability that after traversing x , an atom will suffer a collision within dx in time $dt = pdt = p \frac{dx}{c} = \frac{dx}{\lambda}$ where $\lambda = \frac{c}{p}$ is the free path for atoms with velocity c .

\therefore As total probability = 1, probability of no collision in distance $dx = (1 - \frac{dx}{\lambda})$.

As successive collisions are independent, therefore the joint probability of no collision at $x+dx$ is $f(x) \times (1 - \frac{dx}{\lambda})$

$$\therefore f(x+dx) = f(x) \left(1 - \frac{dx}{\lambda}\right)$$

Expand LHS using Taylor's theorem.

$$f(x) + f'(x)dx + \frac{1}{2} f''(x)(dx)^2 + \dots = f(x) \left(1 - \frac{dx}{\lambda}\right) \quad [\lim_{dx \rightarrow 0}]$$

$$\therefore f'(x) = -f(x)/\lambda. \quad \text{or} \quad \frac{f'(x)}{f(x)} = -\frac{1}{\lambda}$$

$$\text{Integrating, } \ln f(x) = -\frac{x}{\lambda} + \ln C \Rightarrow f(x) = Ce^{-x/\lambda}$$

note that when $x=0$, $f(x)=1$. $\therefore C=1$.

$$\therefore f(x) = e^{-x/\lambda}. \Rightarrow \text{law of distribution of free paths}$$

Method 2

Let, out of N atoms, N' atoms cross x without collision. If after that in dx distance, dN' atoms are thrown out due to collision. Then $\frac{dN'}{dx} \propto N'$ or $\frac{dN'}{dx} = -pN'dx$ (-ive for decrease)

$$\therefore \frac{dN'}{N'} = -pdx$$

$$\text{Integrating, } \ln N' = -px + \ln C$$

$$\text{or } N' = Ce^{-px}. \quad \text{Now put boundary condition at } x=0, N'=N.$$

$$\therefore C=N. \quad \therefore N' = Ne^{-px}$$

$$\text{thrown out molecules are } dN' = +pNe^{-px}dx \quad (+\text{ive number})$$

$$\lambda = \frac{x_1 dN'_1 + x_2 dN'_2 + \dots}{N} = \frac{1}{N} \int_0^\infty x dN'$$

$$= \frac{1}{N} \int_0^\infty x pNe^{-px} dx = p \int_0^\infty e^{-px} x dx$$

$$= p \frac{1}{p^2} \int_0^\infty e^{-x^2/2} x dx = \frac{1}{p} \Gamma(2) = \frac{1}{p}$$

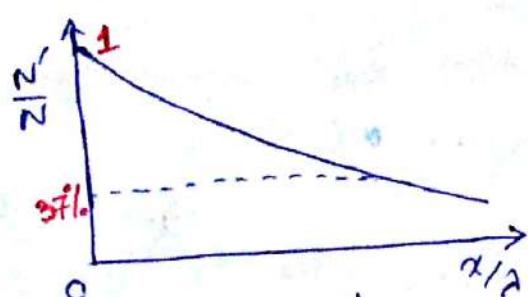
put $px=2$
 $pdx=dz$

$$\therefore \boxed{N' = Ne^{-x^2/2}}$$

$$\therefore f(x) = e^{-x^2/2}$$

This is the "survival equation."

$$dN' = \frac{N}{\lambda} e^{-x/\lambda} dx \quad \therefore \quad \frac{dN'}{dx} = \frac{N}{\lambda} e^{-x/\lambda}$$

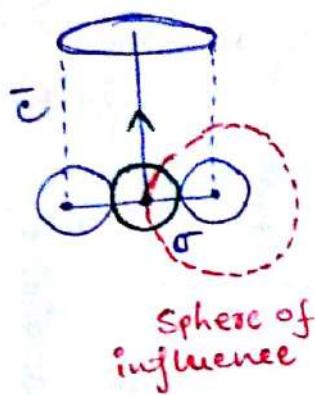


Number $> \lambda$ is $e^{-1} \approx 37\%$
 $< \lambda$ is $1 - 37\% = 63\%$

(a) $P \propto 1/\lambda \Rightarrow$ collision probability is reciprocal of free path.

(b) Intensity of atomic beam \propto number of atoms. $\therefore I' = I e^{-x/\lambda}$.
Initial intensity
↓
Final intensity.

Calculation of λ



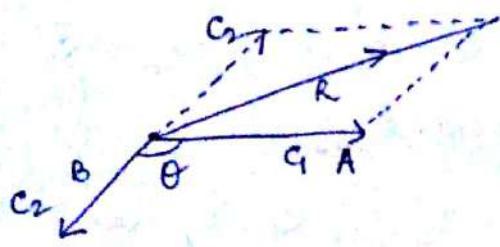
Suppose all atoms are rest but only one moves with velocity \bar{c} . Rigid spherical shape with diameter σ . It can only collide when they touch & can reach distance \bar{c} , so it collides with $\pi \sigma^2 \bar{c} n$ many atoms. This is also number of collisions per second.

$$\therefore \text{Mean free path } \lambda = \frac{\bar{c}}{\pi \sigma^2 \bar{c} n} = \frac{1}{n \pi \sigma^2}$$

This is approximate & Clausius did the first correction followed by Maxwell-Tait.

Clausius correction

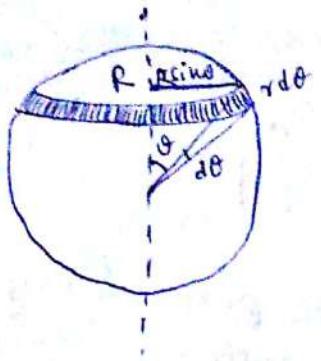
as all atoms are in motion.



Consider A & B atom moves with velocity c_1 & c_2 & angle θ . Making atom B observer (meaning applying equal & opposite velocity c_2 to B), B is in rest

& relative to that A moves with relative velocity

$$R = \sqrt{c_1^2 + c_2^2 - 2c_1 c_2 \cos \theta}$$



Now we have to find mean relative velocity of atom A with respect to all others. If $dN_{\theta,\phi}$ is the number of atoms moving between $\theta \text{ & } \theta + d\theta$, $\phi \text{ & } \phi + d\phi$ then

$$dN_{\theta,\phi} = \frac{N}{4\pi R^2} R^2 \sin\theta d\theta d\phi = \frac{N \sin\theta d\theta d\phi}{4\pi}$$

$$\text{and } \bar{R} = \frac{\int R dN_{\theta,\phi}}{\int dN_{\theta,\phi}} = \frac{\int_0^\pi \int_0^{2\pi} \sqrt{c_1^2 + c_2^2 - 2c_1 c_2 \cos\theta} \frac{N \sin\theta d\theta d\phi}{4\pi}}{\int_0^\pi \int_0^{2\pi} \frac{N \sin\theta d\theta d\phi}{4\pi}}$$

$$= \frac{N}{4\pi} \cancel{\int_0^\pi} \int_0^\pi \sqrt{c_1^2 + c_2^2 - 2c_1 c_2 \cos\theta} \sin\theta d\theta / \frac{N}{4\pi} \cancel{\int_0^\pi}$$

$$\text{substitute } c_1^2 + c_2^2 - 2c_1 c_2 \cos\theta = z^2 \quad \text{or} \quad 2c_1 c_2 \sin\theta d\theta = dz$$

$$\int z^2 \frac{1}{2c_1 c_2} dz = \frac{1}{2c_1 c_2} \int_{(c_1 - c_2)^2}^{(c_1 + c_2)^2} z^2 dz = \frac{1}{2c_1 c_2} \left[\frac{z^3}{3/2} \right]_{(c_1 - c_2)^2}^{(c_1 + c_2)^2}$$

$$= \frac{1}{3c_1 c_2} \left[(c_1 + c_2)^3 - (c_1 - c_2)^3 \right]$$

$$\therefore \bar{R} = \frac{1}{6c_1 c_2} \left[(c_1 + c_2)^3 - (c_1 - c_2)^3 \right].$$

According to Clausius's assumption $c_1 = c_2 = \bar{c}$

$$\therefore \bar{R} = \frac{1}{6\bar{c}^2} 8\bar{c}^3 = \frac{4}{3}\bar{c}, \text{ meaning in traveling a distance } \bar{c}, \text{ number of collision by molecule A with relative velocity}$$

$$\bar{R} \text{ is } \pi \sigma^2 \bar{R} n \text{ & therefore } \lambda_{cl} = \frac{\bar{c}}{\pi \sigma^2 n \bar{R}} = \frac{3}{4} \frac{1}{n \pi \sigma^2}.$$

Maxwell's correction

Clausius took $c_1 = c_2 = \dots = c_N = \bar{c}$

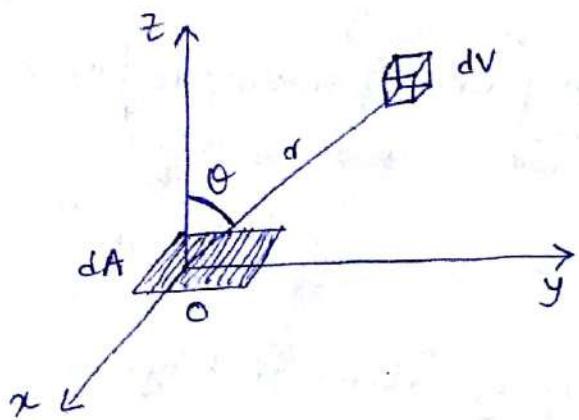
but they're Maxwellian distributed in reality!

Maxwell corrected by considering both $c_1 > c_2$ & $c_1 < c_2$ case
with $dN_{c_2} = 4\pi N \left(\frac{m}{2\pi k_B T} \right)^{3/2} e^{-mc_2^2/2k_B T} c_2^2 dc_2$

to obtain $\bar{R} = \sqrt{2} \bar{c}$ (see AB Gupta § 2.21.2 for derivation)

$$\therefore \lambda_{\text{Maxwell}} = \frac{c}{\sqrt{2\pi} n c} = \frac{1}{\sqrt{2\pi} n}$$

Pressure of a gas using mean free path



Once again, we want to compute atoms within volume dV at distance r with inclination θ to a surface dA at origin that reach dA after collision with other atoms, using survival equation.

Number of molecules between c & $c+dc$ in volume dV is $dn_c dV$. If λ is mean free path of the gas atoms then ~~no~~ number of collision suffered by one atom per unit time = $\frac{c}{\lambda}$.

As $1 \rightarrow 2$ & $2 \rightarrow 1$ collision is counted twice, so the number of collisions suffered by $dn_c dV$ number of atoms in unit time is $\frac{1}{2} \frac{c}{\lambda} dn_c dV$. But each collision results to two new paths along which atoms travel.

∴ The number of new paths or number of atoms emanating from dV per unit time = $\frac{1}{2} \frac{c}{\lambda} dn_c dV \times 2$ & that are pointed towards the area dA is the solid angle subtended by dA at dV = $\frac{dA \cos \theta / r^2}{4\pi}$ (4π = all molecules contained)

∴ That exit from dV pointing to dA , that number is ~~small~~

$$N_0 = \frac{c}{\lambda} dn_c dV \frac{dA \cos \theta}{4\pi r^2} \quad (\text{per unit time})$$

In N_0 , only those atoms with $\lambda \geq r$ can reach dA , which is

$$N = N_0 e^{-r/\lambda} = \frac{c}{\lambda} dn_c dV \frac{dA \cos \theta}{4\pi r^2} e^{-r/\lambda}$$

$$\begin{aligned}
 &= \frac{cdn_c dA \cos \theta \sin^2 \theta \sin \phi dr e^{-r/\lambda}}{\lambda^4 \pi r^2} \\
 &= \frac{dA}{4\pi} \times \frac{cdn_c \sin \theta \cos \theta d\theta d\phi}{4\pi} \times \frac{e^{-r/\lambda}}{\lambda} dr \\
 \therefore \text{No. of atoms striking } dA &= \frac{dA}{4\pi} \int_{c=0}^{\infty} \int_{\theta=0}^{\pi/2} \int_{\phi=0}^{2\pi} \int_{r=0}^{r/\lambda} e^{-r/\lambda} dr
 \end{aligned}$$

$$= \frac{dA}{4\pi} n \bar{c} \frac{1}{2} 2\pi \times 1 = \frac{dA}{4} n \bar{c}$$

So per unit area per unit time, number of atoms striking

$$\therefore \boxed{\frac{n \bar{c}}{4}}$$

Again, we know one atom suffers momentum change = $2mc \cos \theta$

So change of momentum for all atom are

$$\frac{dA}{4\pi} \int_{c=0}^{\infty} 2mc^2 dn_c \int_{\theta=0}^{\pi/2} \sin \theta \cos^2 \theta d\theta \int_{\phi=0}^{2\pi} d\phi \int_{r=0}^{\infty} e^{-r/\lambda} d(r/\lambda)$$

$$dF = \frac{mdA}{2\pi} n \bar{c} \frac{1}{m} 2\pi = \frac{1}{3} m n \bar{c}_{rms}^2 dA$$

$$\therefore \rho = \frac{dF}{dA} = \boxed{\frac{1}{3} m n \bar{c}_{rms}^2}$$

$$\left[\bar{c}_{rms}^2 = \frac{1}{n} \int_{c=0}^{\infty} c^2 dn_c \right]$$

Mean free path of a mixture of a gas

If we consider two different molecule with diameter σ_1, σ_2 then σ_1 diameter molecule will collide with all molecule that are $\frac{\sigma_1 + \sigma_2}{2}$ distance apart from σ_1 molecule. Hence λ will be $1/n\pi\sigma_a^2$ where $\sigma_a = \frac{\sigma_1 + \sigma_2}{2}$ & m = number of molecules per unit volume of σ_2 type. But σ_2 molecules are not at rest then if σ_1 type moves with \bar{c}_1 & σ_2 type moves with \bar{c}_2 & if the molecules of σ_2 move perpendicular to σ_1 then

relative velocity $R = \sqrt{\bar{c}_1^2 + \bar{c}_2^2}$ & therefore $\frac{R}{\bar{c}_1} = \frac{\sqrt{\bar{c}_1^2 + \bar{c}_2^2}}{\bar{c}_1}$

So λ_1 of σ_1 type of molecules within σ_2 type molecules are

$$\lambda_1 = \frac{1}{n\pi\sigma_a^2 \sqrt{\bar{c}_1^2 + \bar{c}_2^2}}$$

Similarly, λ_2 of σ_2 type of molecules

$$\text{within } \sigma_1 \text{ type molecules are } \lambda_2 = \frac{1}{n\pi\sigma_a^2 \sqrt{\bar{c}_1^2 + \bar{c}_2^2}}.$$

perpendicular directionality assumption gives

Maxwell's distribution with relative velocity R , & if we had assumed

$\bar{c}_1 = \bar{c}_2$ & then we could get back Maxwell's expression of free path.

If we now consider n_1 molecule of σ_1 type with \bar{c}_1 & n_2 molecule of σ_2 type with \bar{c}_2 avg. velocity then no. of impact / sec by σ_1

$$\Gamma_1 = \underbrace{\sqrt{2}\bar{c}_1 n_1 \pi \sigma_1^2}_{\sigma_1 \text{ with } \sigma_1} \left(= \frac{\bar{c}_1}{\lambda_{11}} \right) + \underbrace{n_2 \pi \sigma_a^2 \bar{c}_1 \sqrt{\bar{c}_1^2 + \bar{c}_2^2}}_{\sigma_1 \text{ with } \sigma_2} \left(= \frac{\bar{c}_1}{\lambda_{12}} \right)$$

\therefore Mean free path of σ_1 type molecules in the gas mixture

$$\lambda_1 = \frac{\bar{c}_1}{\Gamma_1} = \frac{\bar{c}_1}{\sqrt{2}\pi n_1 \bar{c}_1 \sigma_1^2 + \pi \sigma_a^2 n_2 \sqrt{\bar{c}_1^2 + \bar{c}_2^2}} \text{ & Mean free path for the other}$$

$$\lambda_2 = \frac{\bar{c}_2}{\Gamma_2} = \frac{\bar{c}_2}{\sqrt{2}\pi n_2 \bar{c}_2 \sigma_2^2 + \pi \sigma_a^2 n_1 \sqrt{\bar{c}_1^2 + \bar{c}_2^2}}$$

HW 1. Estimate the size of a He atom, assuming its mean free path

is 28.5×10^{-6} cm at N.T.P. & density is 0.178 gm/litre at N.T.P.

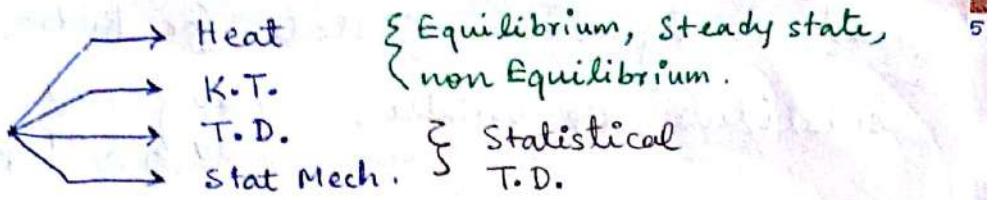
& the mass of He atom is 6×10^{-24} gm.

2. The diameter of a gas molecule is 3×10^{-8} cm. Calculate the mean free path at N.T.P. Given $K_B = 1.38 \times 10^{-16}$ ergs/ $^\circ$ C.

3. Find the diameter of a molecule of Benzene if its mean free path is 2.2×10^{-8} m & the number of Benzene molecules/unit volume is 2.79×10^{25} molecules/m³.

Thermodynamics

Thermal Physics



Useful Mathematical Tools

(a) Partial differentiation : $\frac{\partial f}{\partial a_i}(a_1, a_2, \dots, a_n)$ a_1, \dots, a_n = independent variables.

Let $z = f(x, y)$ is an explicit function (surface plot in XYZ plane)
Motion of a coordinate point on the surface \rightarrow 3 choices (i) $x = \text{constant}$
 y varies (ii) x varies $y = \text{constant}$, (iii) both x, y varies.

$x \rightarrow x + dx$, $y = \text{constant}$, $z = f(x+dx, y)$ from $f(x, y)$.

$$\left(\frac{\partial f}{\partial x}\right)_y = \lim_{dx \rightarrow 0} \frac{f(x+dx) - f(x)}{dx} = f_x$$

$$\text{Similarly } f_y = \left(\frac{\partial f}{\partial y}\right)_x = \lim_{dy \rightarrow 0} \frac{f(x, y+dy) - f(x, y)}{dy}$$

& Higher order derivatives, $f_{xx} = \frac{\partial^2 f}{\partial x^2}$, $f_{xy} = f_{yx} = \frac{\partial^2 f}{\partial x \partial y}$ etc.

(b) Total differentials :

Let $z = f(x, y)$ an explicit function where, x, y are independent.
means $dz \rightarrow 0$ implies $dx \rightarrow 0$ & $dy \rightarrow 0$ independently. Then
 dz is the total differential $dz = \left(\frac{\partial f}{\partial x}\right)_y dx + \left(\frac{\partial f}{\partial y}\right)_x dy$.

If an implicit function, say $f(x, y, z) = 0$ then

$$df = 0 = \left(\frac{\partial f}{\partial x}\right)_{y,z} dx + \left(\frac{\partial f}{\partial y}\right)_{x,z} dy + \left(\frac{\partial f}{\partial z}\right)_{x,y} dz = f_x dx + f_y dy + f_z dz$$

If $dx = 0$, $\left(\frac{\partial y}{\partial z}\right)_x = -\frac{f_z}{f_y}$, If $dy = 0$, $\left(\frac{\partial z}{\partial x}\right)_y = -\frac{f_x}{f_z}$ &

If $dz = 0$, $\left(\frac{\partial x}{\partial y}\right)_z = -\frac{f_y}{f_x}$. $\therefore \left(\frac{\partial x}{\partial y}\right)_z \left(\frac{\partial y}{\partial z}\right)_x \left(\frac{\partial z}{\partial x}\right)_y = -1$.

Also $dz = \left(\frac{\partial f}{\partial x}\right)_y dx + \left(\frac{\partial f}{\partial y}\right)_x dy = M dx + N dy$ is perfect differential

$$\text{if } \left(\frac{\partial M}{\partial y}\right)_x = \left(\frac{\partial N}{\partial x}\right)_y \Rightarrow \frac{\partial^2 z}{\partial y \partial x} = \frac{\partial^2 z}{\partial x \partial y}$$

Example Equation of state for hydrostatic system $f(P, V, T) = 0$
 substitute the variables, $\left(\frac{\partial P}{\partial V}\right)_T \left(\frac{\partial V}{\partial T}\right)_P \left(\frac{\partial T}{\partial P}\right)_V = -1$

$$\beta = \frac{1}{V} \left(\frac{\partial V}{\partial T}\right)_P \quad (\text{volume expansivity})$$

$$\kappa = -\frac{1}{V} \left(\frac{\partial V}{\partial P}\right)_T \quad (\text{isothermal compressibility})$$

$$\therefore \left(\frac{\partial P}{\partial T}\right)_V = - \left(\frac{\partial P}{\partial V}\right)_T \left(\frac{\partial V}{\partial T}\right)_P = + \frac{1}{\kappa V} PV = \frac{\beta}{\kappa}$$

Get back the ~~same~~ expression $P = P(V, T)$

$$\Rightarrow dP = \left(\frac{\partial P}{\partial V}\right)_T dV + \left(\frac{\partial P}{\partial T}\right)_V dT = -\frac{1}{\kappa V} dV + \frac{\beta}{\kappa} dT$$

(c) Line integral & exact differential :

$$dz = M dx + N dy \rightarrow \text{required } z(x, y) \text{ at } (x_1, y_1) \& (x_2, y_2)$$

means $\int_{x_1}^{x_2} M(x, y) dx$, $\int_{y_1}^{y_2} N(x, y) dy$ be evaluated, provided

$y = f(x)$ dependency is given, meaning path in XY plane is given.
 path dependent integration !! Each $f(x)$ gives different result.

However if dz is total differential, then

$$M = \frac{\partial z}{\partial x}, \quad N = \frac{\partial z}{\partial y} \Rightarrow \frac{\partial M}{\partial y} = \frac{\partial^2 z}{\partial x \partial y} = \frac{\partial N}{\partial x}$$

from this we can evaluate z by $\iint_{x,y}$ without using $y = f(x)$
 path independent integration !!

It only depends on (x_1, y_1) & (x_2, y_2) ; z is called "point function".

If contour integral over complete cycle $\oint dz = 0$ if dz
 is exact or total differential.

Change of state of a system may be of different types:

Isothermal: If the change of state is such that the temperature (diathermic) of the system remains constant, then that state is called isothermal ($T = \text{constant}$).

Isobaric: If the process is such that the pressure remains constant then it is called isobaric ($P = \text{constant}$).

Isochoric: If during the change of state, the volume of the system does not change, then it is called isochoric ($V = \text{constant}$)

Adiabatic: If the change is such that there is no exchange of heat then it is called adiabatic ($Q = \text{constant}$)

Isentropic: If during the change, the entropy of the system remains constant, then it is called isentropic ($S = \text{constant}$)

Isenthalpic: If during the change of state, the total heat content remains constant, then it is called isenthalpic process
($H = U + PV = \text{constant}$)

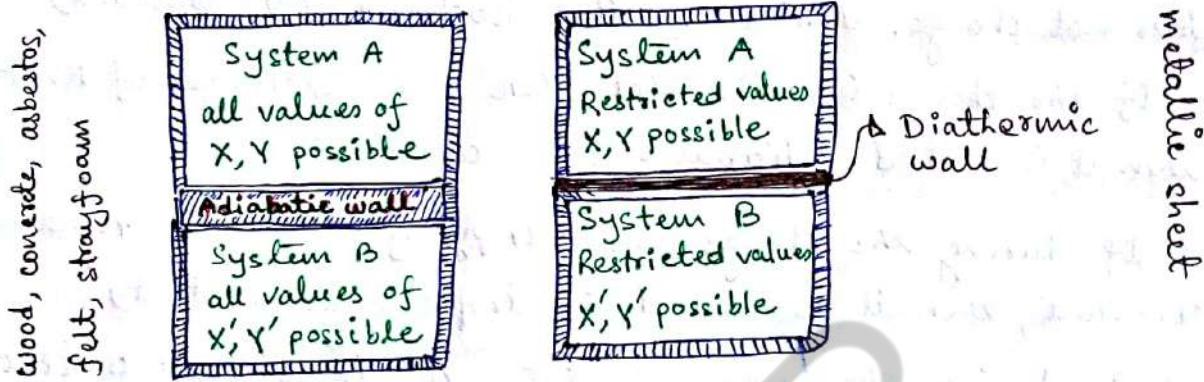
Thermodynamic System "System" refers to a certain portion of the universe within some closed surface (boundary). Boundary may enclose a solid, liquid, gas, collection of magnetic dipoles, portion of liquid surface, batch of radiant energy & so forth. Boundary is not necessarily fixed in shape or size & can be real or imaginary. Like inner surface $\xleftarrow{\text{real}}$ of tank containing a compressed gas or surface $\xleftarrow{\text{imaginary}}$ enclosing certain mass of fluid.

Many problems involve interchange of energy between a given system & others. Such other systems that can interchange energy with the system are called "surroundings". System & surrounding together constitute the universe.

When conditions are such that no energy interchange with the surroundings can take place, then the system is said to be "isolated". If no matter can cross the boundary, then its a "closed" system. But if interchange of matter between system & surrounding, then its an "open" system.

Thermal Equilibrium

Consider a system having two independent coordinates X, Y (say pressure, volume or temperature). A state of the system in which X & Y have definite values that remain constant as long the external conditions are not changed is called an equilibrium state. Such equilibrium state in a system depends on the proximity of other systems & also on the nature of the separating wall.

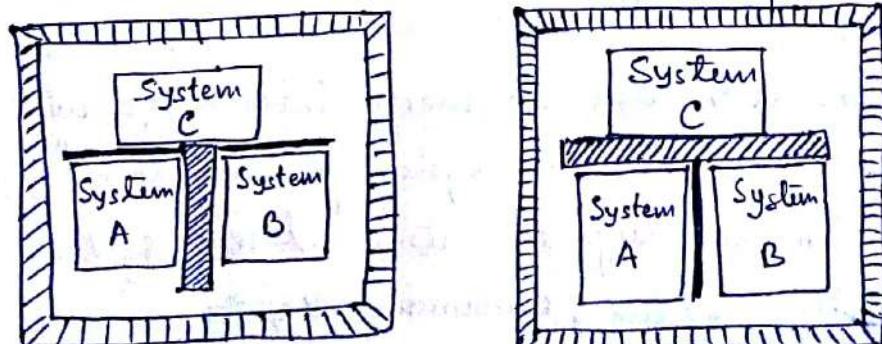


For adiabatic wall, state X, Y for system A may coexist with state X', Y' of system B as equilibrium state. But for diathermic wall, (X, Y) & (X', Y') will change spontaneously until an equilibrium state of the combined system is attained.

Zeroth law of Thermodynamics (Fowler, 1908)

If two system A & B separated by an adiabatic wall but each of them are in contact with a third system C through diathermic wall, then A & B will come to equilibrium with C & no further change will occur if the adiabatic wall between A & B is replaced by diathermic wall.

0th law of T.D. : 2 systems in thermal equilibrium with third are in thermal equilibrium with each other.



Thermodynamic equilibrium

A system is said to be in a state of thermodynamic equilibrium if (a) mechanical equilibrium, (b) chemical equilibrium & (c) thermal equilibrium is satisfied.

When there is no unbalanced force in the interior of a system & also no net force between a system & its surrounding (also net torque is zero), then the system is in a state of mechanical equilibrium.

When a system in mechanical equilibrium does not undergo a spontaneous change in its internal structure (e.g. chemical reaction) mass transfer due to diffusion) then the system is in a state of chemical equilibrium.

Thermal equilibrium exists when there is no spontaneous change in the (thermodynamic) coordinates (e.g. P, V, T) of a system in mechanical & chemical equilibrium when separated from surroundings by diathermic wall.

Non equilibrium states cannot be described in terms of T.D. coordinates as $P = P(x, y, z)$ that varies in space & time.

In the absence of surface, gravitational, electric, magnetic effects a constant mass system that exerts uniform hydrostatic pressure to surroundings is a "hydrostatic system". Categorically 3 systems:

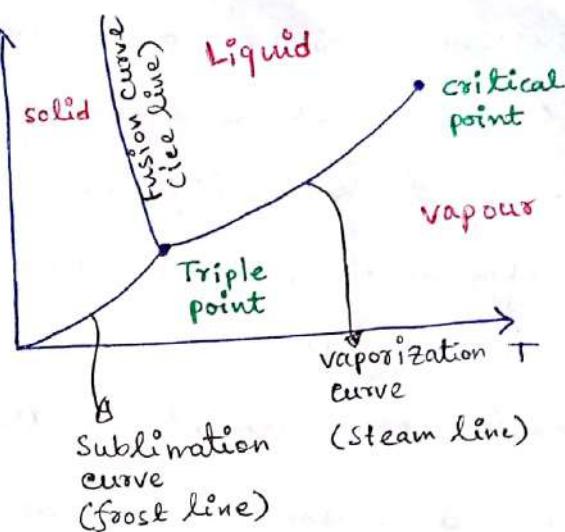
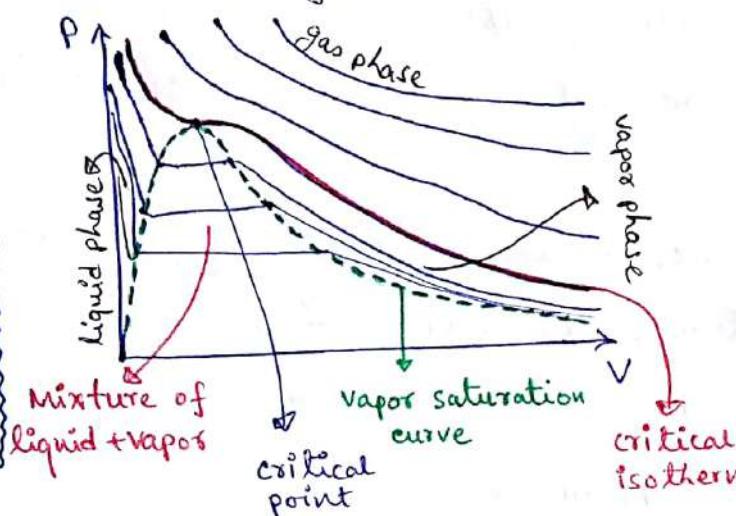
1. A pure substance : 1 chemical constituent (solid, liquid, gas) mixture (s-l, l-g, s-g) or three of them (s-l-g).

2. Homogeneous mixture : mixture of inert gases, mixture of liquids of different constituents (solution)

3. Heterogeneous mixture : mixture of different gases in contact with a mixture of different liquids.

PV & PT diagram for pure substance

INDICATOR DIAGRAM



Thermodynamic description of system other than $f(P, V, T) = 0$

Thermodynamics of a gaseous system is described by three thermodynamic coordinates P, V, T , but for other systems require different types of coordinates.

Stretched Wire $P, V \approx$ unchanged. Thermodynamically equivalent coordinates (a) Tension in the wire (\mathcal{F}), (b) Length of the wire (L) (c) Ideal gas temperature (T). In S.I. units, \mathcal{F} is Newton, L is metre, T in $^{\circ}\text{K}$.

Equivalent of equation of state $f(P, V, T) = 0$ is the Hooke's law at constant temperature within elastic limit.

$$\mathcal{F} = K(L - L_0), \quad L_0 = \text{length at no tension}$$

$$\text{So, } L = L(\mathcal{F}, T) \quad \& \text{ for infinitesimal change}$$

$$\therefore dL = \left(\frac{\partial L}{\partial \mathcal{F}} \right)_T d\mathcal{F} + \left(\frac{\partial L}{\partial T} \right)_{\mathcal{F}} dT$$

$$\text{Linear expansivity } \alpha = \frac{1}{L} \left(\frac{\partial L}{\partial T} \right)_{\mathcal{F}}$$

$$\text{Isothermal Young's modulus } Y = \left(\frac{\partial \mathcal{F}/A}{\partial L/L} \right)_T = \frac{L}{A} \left(\frac{\partial \mathcal{F}}{\partial L} \right)_T$$

$$\therefore dL = \frac{L}{AY} d\mathcal{F} + \alpha L dT.$$

In isothermal condition, all resultants of Young's modulus $dL = 0$,

9

for isothermal condition $dT \approx 0$ & $dL = \frac{L}{AY} dT \approx 0$

therefore work done in expanding a wire is

$$W = \int g dL = \int g \frac{L}{AY} dT = \frac{L}{2AY} g^2.$$

Surface film physical chemistry / chemical engineering / other areas.

- Examples (a) Liquid-vapour interface in equilibrium,
(b) soap bubble/film \rightarrow two surface films with in between liquid.
(c) thin oil film on the surface of water.

Three coordinates (a) surface tension γ (Newton/metre), (b) area of film (A) metre², (c) ideal-gas temperature T . To describe a stretched membrane.

Equation of state $\gamma = \gamma_0 (1 - \frac{T}{T_c})^n$ for surface tension

γ_0 = surface tension at 0°C, $T' \approx$ near T_c & $n \approx 1.22$.

As $T \gg$, $\gamma < \gamma_0$ at $T = T'$.

Dielectric slab

Thermodynamic behaviour of a dielectric slab (whether molecules are polar or nonpolar) is described by three coordinates, (a) electric field intensity E (V/m), (b) electric polarisation (P) in Coulomb/m³ which is the amount of dipole moment per unit volume along the direction of the electric field.

The equation of state of the dielectric

$P = (A + \frac{B}{T})E$ where A, B are constants & depends upon the nature of the dielectric material.

Paramagnetic Rod

When a paramagnetic rod is placed within a solenoid, where the magnetic intensity is H , the rod develops a magnetic moment M . Magnetic induction B in volume V is

$$B = \mu_0 (H + \frac{M}{V}).$$

Equivalent thermodynamic coordinates are (1) Magnetic field intensity H (Ampere/metre), (2) Magnetization M (Ampere metre²), (3) Ideal gas temperature T . (in °K).

The equation of state of thermodynamic equilibrium is

$$M = C \frac{H}{T} \quad (\text{Curie's law})$$

Intensive & Extensive Parameters

If a system in equilibrium is divided into two parts, each with equal mass, then those properties of each half of the system that remain same are called intensive & those which become half are called extensive.

Systems	Intensive coordinates	Extensive coordinates
(a) Hydrostatic system	Pressure (P)	Volume (V)
(b) stretched wire.	Linear tension (F)	Length (L)
(c) surface film	Surface tension (S)	Area (A)
(d) Dielectric slab	Electric field Intensity (E)	Polarization (P)
(e) Paramagnetic rod	Magnetic field Intensity (H)	Magnetization (M)

Equation of state for adiabatic process

Let us consider an ideal gas system of volume V at pressure P . Then for isothermal change, $PV = nRT = \text{constant}$ (as $T = \text{constant}$)

Now for an adiabatic process without any exchange of heat, using first law of thermodynamics $Q = dU + PdV$.

where dU is the internal energy change & if the change in temperature for one mole of gas is dT , then $dU = C_V dT$ where C_V is the specific heat at constant volume.

$\therefore C_V dT + PdV = 0$ for adiabatic process.

from equation of state $PV = RT \therefore PdV + VdP = RdT$

$$\therefore dT = \frac{PdV + VdP}{R} = \frac{PdV + VdP}{C_p - C_v} \text{ as } C_p - C_v = R \text{ for ideal gas}$$

$$\therefore C_v \left(\frac{PdV + VdP}{C_p - C_v} \right) + PdV = 0$$

$$\therefore \frac{C_v PdV + C_v VdP + C_p PdV - C_v PdV}{C_p - C_v} = 0 \therefore C_v VdP + C_p PdV = 0$$

$$\therefore \frac{dp}{P} + \frac{C_p}{C_v} \frac{dv}{v} = 0, \text{ integrating } \ln P + \gamma \ln v = \text{constant}$$

$$\therefore PV^\gamma = \text{constant}$$

$$\text{Using } PV = RT \therefore v = \frac{RT}{P} \Rightarrow P \left(\frac{RT}{P} \right)^\gamma = \text{constant}$$

$$\therefore P^{1-\gamma} T^\gamma = \text{constant}$$

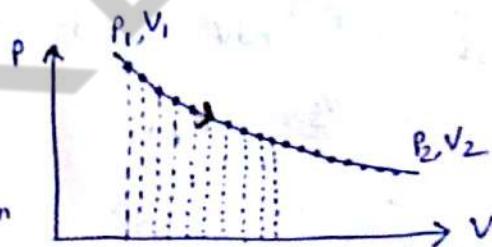
$$\text{Again using } PV = RT, P = \frac{RT}{V} \Rightarrow \frac{RT}{V} v^\gamma = \text{constant}$$

$$\therefore V^{\gamma-1} T = \text{constant}$$

Quasistatic Process

A finite unbalanced force may cause a system to pass through nonequilibrium states. Thus during a process if it is required to describe every state of the system by means of thermodynamic coordinates, the process should not conceive a finite unbalanced force; because a nonequilibrium state cannot be defined by thermodynamic coordinates. Therefore we think of an ideal situation in which external forces vary slightly so as the unbalance force is infinitesimal. A process performed in such ideal way is "quasistatic". [In stat mech course, rethink about "local equilibrium"]

During a quasistatic process the system is infinitesimally near a state of thermodynamic equilibrium, & all states through which the system passes can be described with equation of state with thermodynamic coordinates.



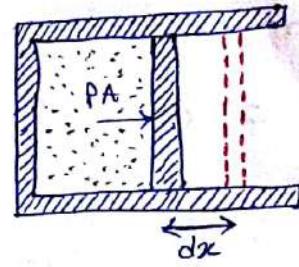
Workdone "on" or "by" a system in a quasistatic process

a) Gaseous hydrostatic System

pressure exerted by the system on piston = P_A

opposing force on the system by external agent

moves piston by dx in opposite direction to P_A



$$\text{Then } dW = -P_A dx = -P dV \quad (dV = Adx)$$

If $dV > 0$ (expansion), $dW < 0$ (negative work)

$dV < 0$ (compression), $dW > 0$ (positive work)

$$\text{for finite quasistatic process } W = - \int_{V_i}^{V_f} P dV$$

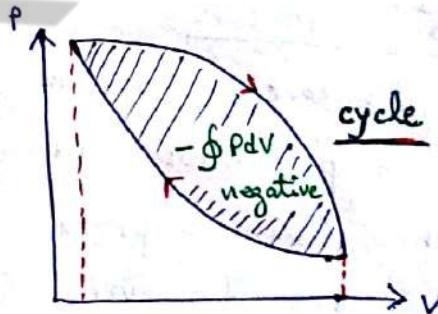
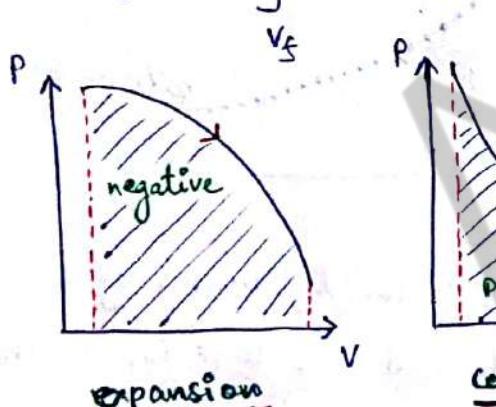
This depends on path because $P = P(T, V) \Rightarrow P = P(V)$ for given T .

∴ Workdone "on" a system from larger volume (V_i) to smaller volume

(V_f) is $W_{if} = - \int_{V_i}^{V_f} P dV$ & for expansion, workdone "by" system

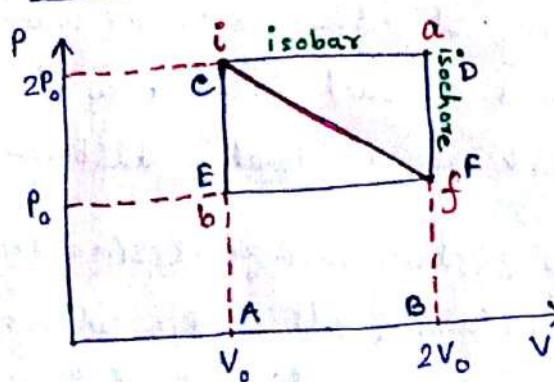
is $W_{fi} = - \int_{V_f}^{V_i} P dV$ and when path is quasistatic, $W_{fi} = -W_{if}$

INDICATOR DIAGRAM



net workdone is negative

Path dependency: Exact & inexact differentials



Many route to go from $i \rightarrow f$.

a) $i \rightarrow a$ (isobaric) $\Rightarrow a \rightarrow f$ (isochoric)

$$\text{W.D.} = - \int P dV = - 2P_0 V_0$$

area ABDC

b) $i \rightarrow b$ (isochoric) $\Rightarrow b \rightarrow f$ (isobaric)

$$\text{W.D.} = - \int P dV = - P_0 V_0$$

area ABFE

$$\textcircled{c} \quad i \rightarrow f, \text{ W.D.} = - \int_{\text{area ABFC}} P dV = - \int_{\text{area ABEF}} P dV - \int_{\text{area EFC}} P dV = - \frac{3}{2} P_0 V_0$$

Infinitesimal amount of work is "inexact differential", means it is not the differential of a function of thermodynamic coordinates. That's represented with dW , & it depends on the path.

Suppose if functional differential $df = 2xy^3 dx + 3x^2y^2 dy$
 $= d(x^2y^3)$.

$\int_{x=y=3}^{x=y=1} df$ depends only on the limit & not on path. df is an "exact differential".

Isothermal quasistatic expansion/compression

$$\text{Using } PV = nRT, \quad W = - \int_{V_i}^{V_f} P dV = - \int_{V_i}^{V_f} \frac{nRT}{V} dV = -nRT \ln \frac{V_f}{V_i}$$

$n = 2 \text{ kmol}$, $T = 273K (0^\circ\text{C})$, $R = 8.31 \text{ kJ/kmol}\cdot\text{K}$, $V_i = 4m^3$, $V_f = 1m^3$
 (compression), $W = 6300 \text{ kJ}$ = positive work (work done "on" gas)

Isothermal quasistatic increase of pressure on solid

$$V = V(P, T) \quad \therefore dV = \left(\frac{\partial V}{\partial P}\right)_T dP + \left(\frac{\partial V}{\partial T}\right)_P dT = \left(\frac{\partial V}{\partial P}\right)_T dP$$

$$\text{isothermal compressibility} = \frac{1}{\text{bulk modulus}} = \frac{1}{dP/(dV/V)} \quad (\text{isothermal})$$

$$\kappa_V = - \frac{1}{V} \left(\frac{\partial V}{\partial P}\right)_T$$

$$\therefore dV = - \kappa_V dP$$

$$\text{so Work done} = \int_{P_i}^{P_f} P \kappa_V dP \approx \frac{\kappa_V}{2} (P_f^2 - P_i^2) = \frac{\kappa m}{2P} (P_f^2 - P_i^2)$$

for copper, at $T = 273K$, $\rho = 8930 \text{ kg/m}^3$, $\kappa = 7.16 \times 10^{-12} \text{ Pa}^{-1}$, $m = 100 \text{ kg}$, $P_i = 0$, $P_f = 1000 \text{ atm} = 1.013 \times 10^8 \text{ Pa}$

$$W = 0.411 \text{ kJ.} = \text{positive work (work done "on" copper)}$$

(b) Isothermal stretching a wire

If tension F changes length of wire from L to $L+dL$ (extension)
then work done on the wire (positive work) is

$$W = \int_{L_i}^{L_f} F dL, \quad \text{remember, } L = L(F, T) \text{ for isothermal}$$

$$dL = \left(\frac{\partial L}{\partial F} \right)_T dF + \left(\frac{\partial L}{\partial T} \right)_F dT = \left(\frac{\partial L}{\partial F} \right)_T dF$$

$$= \int_F^F \frac{L}{AY} F dF \quad \text{using "isothermal Young's modulus"}$$

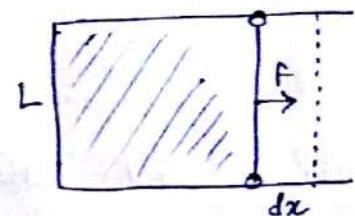
$$= \frac{L}{2AY} F^2.$$

(c) Surface film

If σ is surface tension of a double surface film with liquid in between then force exerted on both film is $2\sigma L$ & for displacement dx , work done "on" the film (positive)

$$dW = 2\sigma L dx = \sigma dA \quad (\text{as } dA = 2L dx)$$

$$\therefore W = \int_{A_i}^{A_f} \sigma dA \quad [\text{For soap bubble } A = 2 \times 4\pi R^2, W = 8\pi \sigma R^2]$$



(d) Polarization of a dielectric solid

Consider a slab of isotropic dielectric material between conducting plates of a parallel-plate capacitor, with area A & separation l connected to a battery to yield potential difference E .

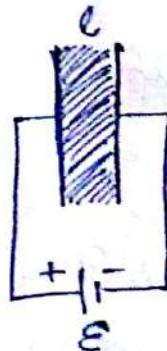
$$\text{Uniform field intensity } E = \frac{V}{l}.$$

Work done against the electric field to move dz amount of charge

$$\therefore dW = Edz = Edz = EdA dD = EV dD \quad [\text{as } dA = DA = \text{total charge} \\ \& D = \text{electric displacement}]$$

$$\text{Now } D = \epsilon E = \epsilon_0 E + \frac{\rho}{V}$$

$$[\text{as } V = EA]$$



$$\therefore dD = (\epsilon_0 dE + \frac{dP}{V}) \quad \text{& then}$$

$$dW = EV(\epsilon_0 dE + \frac{dP}{V}) = V\epsilon_0 E dE + EdP$$

[vacuum] [material]

\therefore Net work on dielectric is $dW = EdP$, $W = \int_{P_i}^{P_f} EdP$

(e) Magnetization of a magnetic solid

magnetic ring cross sectional area A,
circumference L. Insulated wire is wind
on that (Toroidal winding)

Current in the winding initiate magnetic
field with induction B (uniform). Using
Rheostat if current is changed in time dt then

$$E = -NA \frac{dB}{dt} \quad (\text{Faraday's principle of EM induction})$$

$$N = \text{number of turns} \quad H = ni = \frac{Ni}{L} = \frac{(NAi)}{V} \quad (V=AL)$$

If dZ charge is transferred in circuit, work done "by" system
to maintain current is $dW = -E dZ = NA \frac{dB}{dt} dZ$

$$= NA \frac{dZ}{dt} dB = \frac{(NAi)}{V} dB$$

$$= VH dB.$$

If M = total magnetic moment, then

$$B = \mu H = \mu_0 H + \mu_0 \frac{M}{V} \Rightarrow dB = \mu_0 dH + \mu_0 \frac{dM}{V}$$

$$\therefore dW = V\mu_0 H dH + \mu_0 H dM$$

[vacuum] [material]

Work done to change the magnetization is $dW = \mu_0 H dM$

$$\therefore W = \mu_0 \int_{M_i}^{M_f} H dM.$$

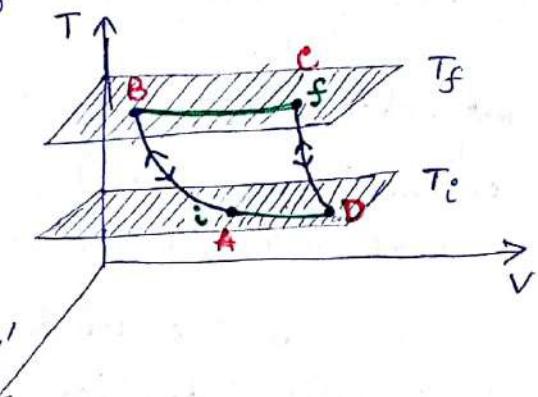
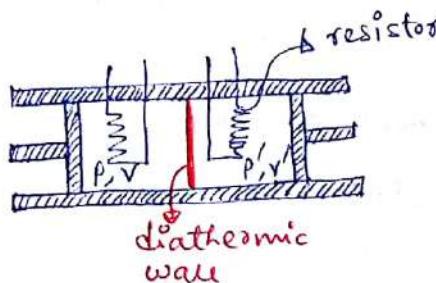
$$\boxed{\text{Work (extensive)} = \text{Intensive quantity} \times \text{Extensive quantity}}$$

Using Curie's law $M = C \frac{H}{T}$

$$W = \mu_0 \int_{H_i}^{H_f} \frac{MT}{C} dM = \frac{\mu_0 T}{2C} (M_f^2 - M_i^2)$$

$$= \mu_0 \int_{H_i}^{H_f} H \frac{C}{T} dH = \frac{\mu_0 C}{2T} (H_f^2 - H_i^2)$$

Adiabatic Work



System can undergo adiabatic work with surroundings @ moving pistons quasistatically (slowly) $W = - \int P dV$

- ⑥ non-quasistatic (free expansion) \Rightarrow fast so that velocity of molecules do no work at piston. Also work can be done by dissipating electrical energy on resistors

Path ABC from $i \rightarrow f$, AB is reversible quasistatic adiabatic compression between two isothermal planes. BC correspond to adiabatic irreversible dissipation of electrical energy to keep $T = \text{constant}$, as energy can be added to resistor but cannot be extracted.

There are many such other path (e.g. ADC), meaning adiabatic work is same along all paths.

1st law of Thermodynamics: If a system is changed from initial state to final state by adiabatic work, the work done is same for all adiabatic path connecting two states.

$$W_{i \rightarrow f} (\text{adiabatic}) = U_f - U_i = dU \quad (\text{perfect differential})$$

U = internal energy function

But if system change state nonadiabatically, then to conserve energy of heat is to be added to system (+ive)/subtracted (-ive)

from the system, $U_f - U_i = Q + W$.

for infinitesimal process, $dU = dQ + dW$ & if its quasi static
then dU & dW can be expressed in thermodynamic coordinates.

for hydrostatic system $dU = dQ - PdV$

wire $dU = dQ + \gamma dL$

surface film $dU = dQ + \sigma dA$

paramagnetic Rod $dU = dQ + \mu_0 H dM$

When $U = U(V, T)$, $dU = \left(\frac{\partial U}{\partial T}\right)_V dT + \left(\frac{\partial U}{\partial V}\right)_T dV$

$$\therefore dQ = \left(\frac{\partial U}{\partial T}\right)_V dT + \left[\left(\frac{\partial U}{\partial V}\right)_T + P\right] dV \quad \text{--- (1)}$$

$$\therefore \frac{dQ}{dT} = \left(\frac{\partial U}{\partial T}\right)_V + \left[\left(\frac{\partial U}{\partial V}\right)_T + P\right] \frac{dV}{dT}$$

(i) $V = \text{constant}$, $\left(\frac{dQ}{dT}\right)_V = \left(\frac{\partial U}{\partial T}\right)_V = C_V$

(ii) $P = \text{constant}$, $\left(\frac{dQ}{dT}\right)_P = \left(\frac{\partial U}{\partial T}\right)_V + \left[\left(\frac{\partial U}{\partial V}\right)_T + P\right] \left(\frac{\partial V}{\partial T}\right)_P$

$$\therefore C_P = C_V + \left[\left(\frac{\partial U}{\partial V}\right)_T + P\right] V \beta \quad \beta = \frac{1}{V} \left(\frac{\partial V}{\partial T}\right)_P$$

$$\therefore \left(\frac{\partial U}{\partial V}\right)_T = \frac{C_P - C_V}{V \beta} - P$$

Cyclic process $dU = 0$, $dQ = PdV$. heat = work,

Also, from equation (1) $dQ = C_V dT + \left[\left(\frac{\partial U}{\partial V}\right)_T + P\right] dV$

for ideal gas, no potential energy, free expansion do not affect heat.

$PdV = 0$, $dQ = 0$ at constant temperature $dT = 0$, $\left(\frac{\partial U}{\partial V}\right)_T dV = 0$.

$\therefore \left(\frac{\partial U}{\partial V}\right)_T = 0 \Rightarrow$ internal energy is independent of volume.

$\therefore dQ = C_V dT + PdV$ for ideal gas.

Equation of state $PV = nRT \Rightarrow PdV + VdP = nRdT$ infinitesimal
quasi static process

$\therefore dQ = (C_V + nR)dT - VdP$

$$\therefore \frac{dQ}{dT} = C_V + nR - V \frac{dP}{dT}, P \text{ constant}$$

$$C_P = C_V + nR$$

$$\text{Also } \delta Q = C_p dT - V dP$$

Elasticity of perfect gas

Isothermal bulk modulus $E_T = -V \left(\frac{\partial P}{\partial V} \right)_T$

Adiabatic bulk modulus $E_g = -V \left(\frac{\partial P}{\partial V} \right)_g$.

For isothermal change $PV = RT = \text{constant}$

$$PV + VdP = 0 \Rightarrow \left(\frac{\partial P}{\partial V} \right)_T = -\frac{P}{V}.$$

For adiabatic change $PV^\gamma = \text{constant}$, $\gamma PV^{\gamma-1} dV + dP V^\gamma = 0$

$$\therefore \left(\frac{\partial P}{\partial V} \right)_g = -\frac{\gamma P}{V}. \quad \therefore \boxed{\frac{E_g}{E_T} = \gamma.}$$

Second Law of Thermodynamics

This is an extension of first law with a direction of the process \rightarrow experimental development of engines.

Reversibility & Irreversibility

reversibility = system + surroundings from changed state can restore back to initial configuration without change of universe.
The opposite to that is irreversible process.

2nd law \rightarrow Carnot's finding in ideal engine \rightarrow Clausius statement \rightarrow Kelvin's statement \rightarrow birth of "entropy".

Conversion of work to heat

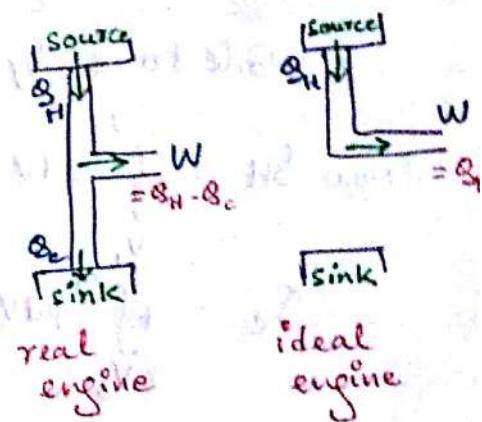
In previous examples, we saw how heat can be generated indefinitely at the expense of work. To study the converse process, there may be 1 or more than 1 (series of) processes.

At first it appeared that isothermal expansion may be suitable for conversion of heat into work, but it's not indefinite. Thus a cycle is required that can be taken back & forth.

If in a cycle Q_H amount of heat is absorbed & Q_C ($Q_C < Q_H$) amount of heat is radiated & W amount of work is needed then the mechanical device is a "Heat Engine." Efficiency of such engine is defined as

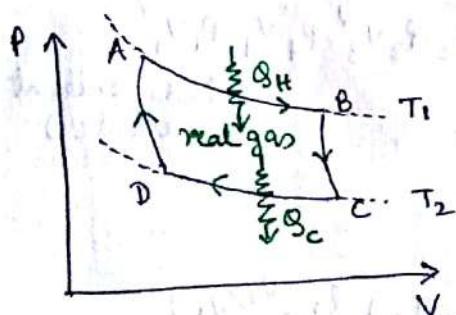
$$\eta = \frac{\text{work output}}{\text{Heat input}} = \frac{W}{Q_H} = \frac{Q_H - Q_C}{Q_H}$$

Hot reservoir of infinite heat capacity is "source" & cold reservoir is called "sink."



Carnot cycle

It's maximum efficient engine (nearly ideal)



AB, CD = isothermal
BC, DA = adiabatic

Consists of four reversible steps.

(1) Isothermal expansion $A \rightarrow B$:



Cylinder is placed in contact with a heat reservoir at temperature T_1 & gas expands isothermally from (P_1, V_1, T_1) to (P_2, V_2, T_1) . Q_H amount of heat taken from source & $-W_1$ amount of work done "by" the gas.

(2) Adiabatic expansion $B \rightarrow C$: The gas is allowed to expand adiabatically in insulated environment. The temperature falls to T_2 & volume changes to V_3 . $Q=0$ and work done "by" the gas is $-W_2$.

(3) Isothermal compression $C \rightarrow D$: The gas at very low pressure is now reversibly compressed in sink at temperature T_2 from volume V_3 to V_4 . Heat rejected to sink is Q_C & work done by the gas is $+W_3$ (positive because work done "on" gas).

(4) Adiabatic compression $D \rightarrow A$: Now the system is adiabatically restored to initial state from $V_4 \rightarrow V_1$, $T_2 \rightarrow T_1$. $Q=0$, work done

by the gas is $+W_1$ (plus because workdone "on" the gas).

from first law of thermodynamics $\oint_C dU = 0$

$$\text{or } \oint_C dQ - \oint_C dW = 0 \text{ or } Q_H + 0 - Q_C + 0 = -W_1 + W_2 + W_3 + W_4$$

∴ Work done by engine = difference of heat absorbed & rejected.

$$\text{Now } Q_H = - \int_{V_1}^{V_2} pdV = -RT_1 \ln \frac{V_2}{V_1} = W_1.$$

$$Q_C = + \int_{V_3}^{V_4} pdV = -RT_2 \ln \frac{V_3}{V_4} = -W_3$$

But for isothermal processes $P_1V_1 = P_2V_2, P_3V_3 = P_4V_4$

for adiabatic processes $P_2V_2^\gamma = P_3V_3^\gamma, P_4V_4^\gamma = P_1V_1^\gamma$

Multiplying, $P_1V_1 P_2V_2^\gamma P_3V_3^\gamma P_4V_4^\gamma = P_2V_2 P_3V_3^\gamma P_4V_4^\gamma P_1V_1^\gamma$ (independent events)

$$\text{or } (V_2 V_1)^{\gamma-1} = (V_3 V_4)^{\gamma-1}$$

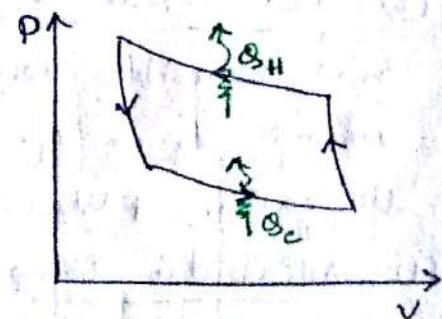
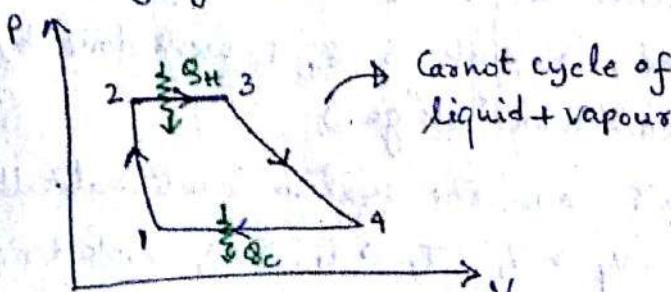
$$\text{or } V_2/V_1 = V_3/V_4$$

$$\therefore \text{Efficiency } \eta = \frac{Q_H - Q_C}{Q_H} = \frac{-R(T_1 - T_2) \ln(V_2/V_1)}{-RT_1 \ln(V_2/V_1)}$$
$$= \frac{T_1 - T_2}{T_1} \neq 100\%$$

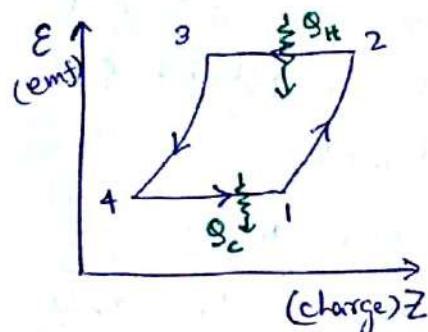
$$\therefore \boxed{\frac{Q_H}{T_1} = \frac{Q_C}{T_2}}$$

We can also reverse the cycle from any point of the indicator diagram

In that case Q_C heat is removed from sink & Q_H is delivered to source & work must be done to the system. The system then works as a refrigerator.



- $1 \rightarrow 2$ reversible adiabatic compression to temperature T_H
 $2 \rightarrow 3$ reversible isothermal isobaric vaporization
 $3 \rightarrow 4$ reversible adiabatic expansion temperature falls to T_C
 $4 \rightarrow 1$ reversible isothermal isobaric condensation



- $1 \rightarrow 2$ reversible adiabatic flow of charge from - to + to temperature T_H .
 $2 \rightarrow 3$ reversible isothermal flow of charge from + to -
 $3 \rightarrow 4$ reversible adiabatic flow of charge to temp. T_C
 $4 \rightarrow 1$ reversible isothermal flow of charge

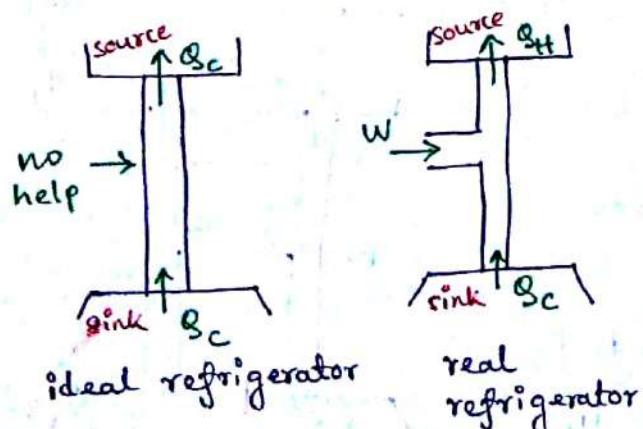
Second law of Thermodynamics

Kelvin's Statement It is impossible by any inanimate object to derive mechanical work from any portion of the matter by cooling it below the temperature of the coldest of the surrounding objects.

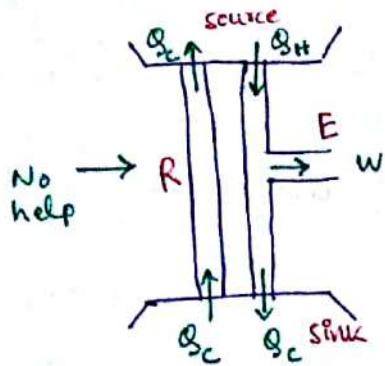
Plank's Statement It is impossible to fabricate an engine working in a complete cycle to produce no effect other than raising a weight & cooling of a hot reservoir.

P-K Statement \rightarrow It is impossible to produce an ideal engine.

Clausius Statement It is impossible for a self-acting machine unaided by any external agency to convey heat from one body at lower temperature to another at a higher temperature. Heat cannot by itself transit from a colder to a warmer body or it is impossible to construct an ideal refrigerator.



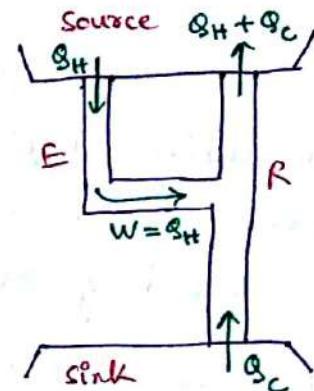
Equivalence of Kelvin-Plank & Clausius statement



Suppose we disobey Clausius's statement & make an ideal refrigerator R that transfers Q_C from sink to source without any work. But engine E draws Q_H from source & returns Q_C to sink & delivering work $Q_H - Q_C$.

∴ Combined $R-E$ system form a self-acting device drawing $Q_H - Q_C$ heat from hot reservoir & convert fully this heat into work without change in sink → violation of K-P statement.

Consider again an ideal engine E that rejects no heat to sink & violate K-P statement. Q_H heat is fully converted to work W which is feeded to a refrigerator R to extract Q_C heat from sink. Again $E-R$ forms a self-acting device that transfers Q_C from sink to source without any change elsewhere → violation of Clausius statement.



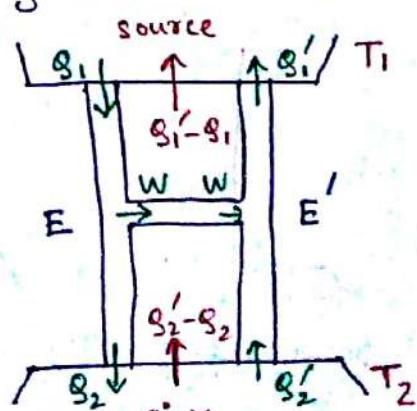
1st law of Thermodynamics = conservation of energy

2nd law of Thermodynamics = directionality (may not be true always)

Carnot's theorem

Efficiency of all reversible engines operating between same two temperatures is equal & no irreversible engine can have a better efficiency compared to the reversible engine between same two temperatures.

Clausius & Kelvin showed that this theorem is a consequence of the 2nd law of thermodynamics. η is independent of working substance & depends only on T_1, T_2 .



Proof Suppose engine E & E' working between T_1 & T_2 differ in working substance or initial pressure or length of stroke. E runs forward & E' run backward (refrigerator). The system is so made that work done by E is exact amount of work needed for E' to function. Suppose η (efficiency) of E $>$ η' of E'.

$\therefore \frac{Q_1 - Q_2}{Q_1} > \frac{Q'_1 - Q'_2}{Q'_1}$. By construction work done per cycle be $w = w'$ $\Rightarrow Q_1 - Q_2 = Q'_1 - Q'_2$

$$\therefore w = w' \Rightarrow Q_1 - Q_2 = Q'_1 - Q'_2$$

$$\therefore \frac{1}{Q_1} > \frac{1}{Q'_1} \quad \text{as } Q'_1 > Q_1 \quad \text{and } Q'_2 > Q_2 \text{ to satisfy } w = w'$$

Therefore hot source gains heat $Q'_1 - Q_1$ and cold sink loses heat $Q'_2 - Q_2$ and no work is done by the combined E + E' system. We've transferred heat from a cold to hot body without performing work is direct contradiction to Clausius statement.

$$\therefore \eta > \eta'$$

Similarly, by reversing the engine E backward & E' forward we can prove that $\eta' > \eta$. So the only possibility is

$$\boxed{\eta = \eta'}$$

for irreversible engine we can show by the same procedure that $\eta_{irr} > \eta_{rev}$ but due to irreversibility we "cannot" show $\eta_{rev} > \eta_{irr}$
 $\therefore \eta_{irr}$ is either equal to or less than η_{rev} .

$$\therefore \boxed{\eta_{rev} \geq \eta_{irr}}$$

Kelvin scale / Absolute scale / Thermodynamic scale of temperature

Using Carnot engine, temperature can be defined in term of energy & the scale so obtained is independent of nature of any particular substance. Efficiency of all reversible engine is a function of two temperatures only

$$\eta = \frac{W}{Q_1} = f(T_1, T_2) = \frac{Q_1 - Q_2}{Q_1} \Rightarrow \frac{Q_1}{Q_2} = \frac{1}{1-f(T_1, T_2)} = F(T_1, T_2).$$

Suppose we have three reversible engines working between the temperature (T_1, T_2) , (T_2, T_3) & (T_1, T_3) , then

$$\frac{Q_1}{Q_2} = F(T_1, T_2), \quad \frac{Q_2}{Q_3} = F(T_2, T_3)$$

$$\therefore \frac{Q_1}{Q_3} = F(T_1, T_2) F(T_2, T_3) = F(T_1, T_3).$$

This can satisfy if & only $F(T_1, T_3) = \frac{\psi(T_1)}{\psi(T_3)}$

∴ For any reversible engine we can write $\frac{Q_1}{Q_2} = \frac{\psi(T_1)}{\psi(T_2)}$

Now if $T_1 > T_2$, $Q_1 > Q_2$ and therefore $\psi(T_1) > \psi(T_2)$

∴ $\psi(T)$ is a monotonically increasing function of temperature

$$\therefore \boxed{\frac{Q_1}{Q_2} = \frac{T_1}{T_2}}$$

So we have defined a new scale of temperature whose ratio depends on heat absorbed & heat rejected in a reversible engine.

$T=0^\circ$ in this scale is when $Q_2=0$ & ∴ $W=Q_1$ & $\eta=100\%$ (ideal engine). T cannot be less than this as $Q_2 < 0$ means engine would be drawing heat both from source & sink which is impossible due to direct violation of 2nd law. ∴ $T=0$ is the lowest attainable temperature when one gets an ideal engine.

Entropy In any reversible process, if heat Q is added at temperature T and rejects Q' heat at temperature T' , then $\frac{Q}{T} = \frac{Q'}{T'}$

$$\therefore \frac{Q}{T} - \frac{Q'}{T'} = 0 \quad \text{or} \quad \sum \frac{Q}{T} = 0 \quad (Q = +\text{ive heat absorbed} \\ = -\text{ive heat rejected})$$

Any reversible transformation is made of a number of infinitesimal reversible isothermal & adiabatic processes. \therefore for any reversible cyclic process ABCD $\oint_C \frac{dQ}{T} = 0$. This is called "Clausius Theorem".

We can define a thermodynamic function "entropy" $dS = \frac{dQ}{T}$ that has the property

$$\oint_{ABCD} \frac{dQ}{T} = 0 \quad \therefore \oint_{ABC} \frac{dQ}{T} + \oint_{CDA} \frac{dQ}{T} = 0$$

$\therefore \oint_{ABC} \frac{dQ}{T} = \oint_{ADC} \frac{dQ}{T}$. This means that $\int \frac{dQ}{T}$ is independent of the path and depends only on the initial & final point 1 & 2.
 $\therefore dS = \frac{dQ}{T}$ is an exact differential. Unit cal/K.

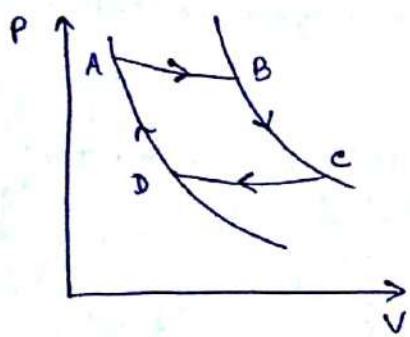
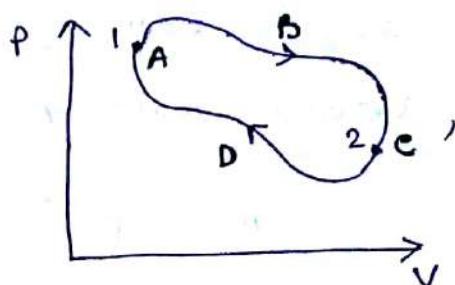
To understand property of dS , consider a reversible cycle constituted by two adiabatic AD & BC & two isothermals AB & CD.

For AD & BC, $dQ = 0 \therefore dS = 0$. & hence

$$\int_A^B \frac{dQ}{T} = \int_D^C \frac{dQ}{T}.$$

Point A & B may be taken anywhere on the adiabatic AD & BC, still the integral will remain same as $dQ = 0$ for adiabatic process. Hence it follows that if we pass from one adiabatic to other, $\int^2 \frac{dQ}{T}$ increases by a definite amount independent of the manner of transformation. We can choose any standard state n & then $\int_n^A \frac{dQ}{T}$

on a reversible path depends only on state n & A. If my standard reference state n is fixed, integral is a function of state A only



and termed as S_A & can be different if we choose different n .

Then the entropy is undetermined to the extent of an additive constant that represents entropy of an arbitrarily chosen standard state with respect to state with zero entropy.

$\therefore S_A = S_n + \int_n^A \frac{ds}{T}$. S_n = entropy at state n which according to law of thermodynamics S_n cannot be determined & absolute value of entropy cannot be found out. But change of entropy between states A & B can be accurately determined,

$$S_A - S_B = S_n + \int_n^A \frac{ds}{T} - S_n - \int_n^B \frac{ds}{T} = \int_B^A \frac{ds}{T}$$

This is valid for equilibrium states only & for nonequilibrium states can approach equilibrium by irreversible process in which entropy increases.

Also $ds = dU + PdV$ is an inexact differential as $\int (dU + PdV)$ requires a knowledge of path. But $ds = \frac{ds}{T}$ is perfect or exact differential.

Exact differential =	Inexact differential \times Integrating factor
(ds)	(ds)
	(YT)

Example $df = 3xydx + 2x^2dy$ = inexact $\frac{\partial^2 f}{\partial x \partial y} \neq \frac{\partial^2 f}{\partial y \partial x}$

But $\underbrace{xy}_{I.F.} df = d\phi = 3x^2y^2dx + 2x^3ydy = d(x^3y^2)$ = exact.

Caratheodory's Principle In the immediate neighbourhood of any arbitrary initial state, there exists neighbouring states which are not accessible from the initial state along adiabatic paths.

Entropy of an ideal gas

If an ideal gas of $m \text{ gm}$ at temperature T & volume V is given heat δQ , then change in entropy is $ms = \int \frac{dU + PdV}{T}$

$$\text{Now } dU = mC_V dT, \quad P = \frac{mRT}{MV}$$

$$\therefore ms = m \left\{ C_V \int_{T_1}^{T_2} \frac{dT}{T} + \frac{R}{M} \int_{V_1}^{V_2} \frac{dV}{V} \right\} \text{ for } (P_1, V_1, T_1) \rightarrow (P_2, V_2, T_2)$$

In general for monoatomic gas $C_V \neq C_V(T)$ & so

$$ms = m \left\{ C_V \ln T + \beta_M \ln V \right\} + \text{constant} \quad \therefore S = S(T, V).$$

To obtain $S = S(T, P)$, we use $C_P - C_V = \beta_M$.

$$\therefore ms = m(C_P - \beta_M) \ln T + \frac{mR}{M} \ln V + \text{constant}$$

$$= mC_P \ln T - \frac{mR}{M} \ln (T_V) + \text{constant} \quad [PV = \frac{mRT}{M}]$$

$$= mC_P \ln T - \frac{m\beta_M}{M} \ln \left(\frac{PM}{mR} \right) + \text{constant} \quad T_V = \frac{PM}{mR}$$

$$ms = mC_P \ln T - \frac{m\beta_M}{M} \ln P + \text{constant}.$$

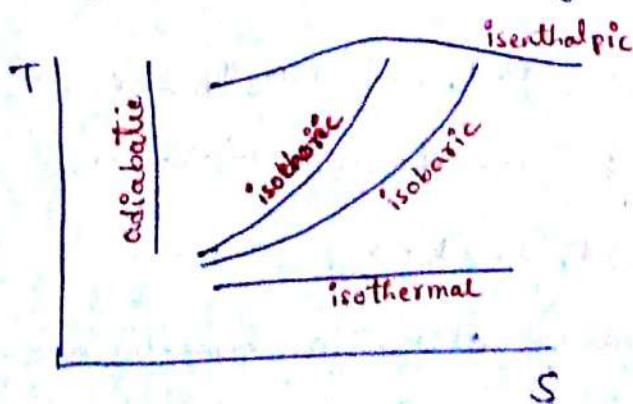
Isothermal system for isothermal change of state of gas $dU = 0$

$$\therefore ds = \frac{pdv}{T} \text{ and using } PV = nRT$$

$$= \frac{nR dv}{V} \quad \therefore S = nR \ln \left(\frac{V_2}{V_1} \right)$$

Adiabatic system for adiabatic change $\delta Q = 0$ & so $ds = 0$. No change in entropy of the system (isentropic process).

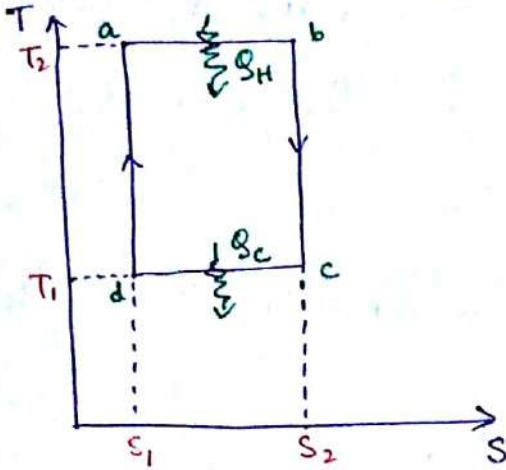
In free expansion of a perfect gas into an empty vessel in a thermally insulated environment, gain in entropy is $\Delta S = \int \frac{pdv}{T} = R \ln \frac{V_2}{V_1}$.



In a reversible procen $Q_R = \int_i^f T ds$

So the heat is the area under a curve in $T-S$ plane.

Carnot's cycle in TS diagram



The Carnot cycle in TS diagram is a parallelogram.

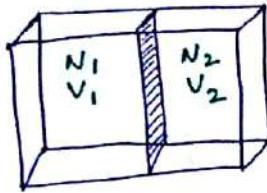
$$Q_H = T_2 (S_2 - S_1)$$

$$Q_C = T_1 (S_2 - S_1) \quad \text{and} \quad dS = 0 \text{ for bc and da lines.}$$

$$\therefore \text{Efficiency } \eta = \frac{W}{Q_H} = \frac{Q_H - Q_C}{Q_H}$$

$$= \frac{(S_2 - S_1)(T_2 - T_1)}{T_2 (S_2 - S_1)} = 1 - \frac{T_1}{T_2}$$

Gibbs' Paradox



Consider an ideal gas of N particles in a container with volume V . A partition divides N_1 & N_2 particles & volume V_1 & V_2 such that

$$N = N_1 + N_2, \quad V = V_1 + V_2 \quad \text{and} \quad n = \frac{N_1}{V_1} = \frac{N_2}{V_2} \quad (\text{equal number density})$$

$$\begin{aligned} \text{Using } S &= C_V \ln T + R \ln V + \text{constant} \\ &= \frac{3}{2} R \ln T + R \ln V + \text{constant} \\ &= \frac{3}{2} N_1 k_B \ln T + N_1 k_B \ln V + \text{constant} \end{aligned}$$

firstly, $T \rightarrow 0, S \rightarrow -\infty !!$ violating 2nd law of thermodynamics

Secondly, $S \propto N$ for a given $V !!$ This expression from classical framework is not right. This also leads to gives paradox \Rightarrow

$$\text{In the partitioned state } S_1 = \frac{3}{2} N_1 k_B \ln T + N_1 k_B \ln V_1 + \text{cont.}$$

$$S_2 = \frac{3}{2} N_2 k_B \ln T + N_2 k_B \ln V_2 + \text{cont.}$$

The partition is now removed & gas comes to well-mixed state.

$$S = \frac{3}{2} (N_1 + N_2) k_B \ln T + (N_1 + N_2) k_B \ln (V_1 + V_2)$$

Now its monoatomic gas of identical atoms, so removing the partition shouldn't increase entropy as particles are indistinguishable.

So $\Delta S = S - (S_1 + S_2) = 0$ But here we find

$$\begin{aligned}\Delta S &= (N_1 + N_2) k_B \ln(V_1 + V_2) + \frac{3}{2} (N_1 + N_2) k_B \ln T - \\&\quad N_1 k_B \ln V_1 - \frac{3}{2} N_1 k_B \ln T - N_2 k_B \ln V_2 - \frac{3}{2} N_2 k_B \ln T \\&= N_1 k_B [\ln(V_1 + V_2) - \ln V_1] + N_2 k_B [\ln(V_1 + V_2) - \ln V_2] \\&= N_1 k_B \ln \frac{V_1 + V_2}{V_1} + N_2 k_B \ln \frac{V_1 + V_2}{V_2} \neq 0.\end{aligned}$$

This paradox cannot be resolved in classical MB statistics & need statistical physics, namely quantum stat mech where atoms are indistinguishable (leads to Sackur-Tetrode formula of entropy in ideal gas) taking Stirling's approximation ($\ln N! = N \ln N - N$). Now, if we correct S accordingly,

$$S = N k_B \ln \left(\frac{V}{N} \right) + \frac{3}{2} N k_B \ln T + \text{constant}$$

$$\begin{aligned}\text{Then } S - (S_1 + S_2) &= (N_1 + N_2) k_B \ln \left(\frac{V_1 + V_2}{N_1 + N_2} \right) - N_1 k_B \ln \frac{V_1}{N_1} - N_2 k_B \ln \frac{V_2}{N_2} \\&= N_1 k_B \ln \frac{V}{N} - N_1 k_B \ln \frac{V_1}{N_1} + N_2 k_B \ln \frac{V}{N} - N_2 k_B \ln \frac{V_2}{N_2} \\&= N_1 k_B \ln \left(\frac{N N_1}{N V_1} \right) + N_2 k_B \ln \left(\frac{V N_2}{N V_2} \right)\end{aligned}$$

$$\text{But } n = \frac{N}{V} = \frac{N_1}{V_1} = \frac{N_2}{V_2} \quad (\text{no. density is constant})$$

$$\therefore \ln \left(\frac{V N_1}{N V_1} \right) = \ln \left(\frac{n}{n} \right) = \ln(1) = 0$$

$$\ln \left(\frac{V N_2}{N V_2} \right) = 0. \quad \therefore \underline{\Delta S = 0}$$

Gibb's paradox thus gets resolved.

Entropy increase in Irreversible process

Consider an irreversible process in which temperature of a body at T_1 is increased to T_2 by coupling to a reservoir at T_2 . The initial & final state of the body is same & its irreversible as heat flow from reservoir to body cannot be reversed. But change in entropy of the body is same for both reversible or irreversible process as its not path dependent.

Suppose pressure remains constant, then change in entropy of the body

$$\therefore \Delta S_{\text{body}} = \int_{T_1}^{T_2} C_p \frac{dT}{T} = C_p \ln(T_2/T_1) > 0 \text{ as } T_2 > T_1.$$

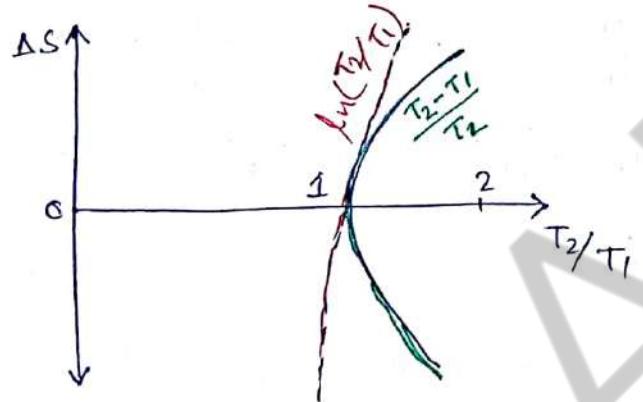
The reservoir temperature T_2 remains constant, so change in entropy is same for reversible isothermal process (heat flowing into reservoir) & irreversible isothermal process (heat only flowing out)

Heat flow into the body $\therefore \delta Q = C_p(T_2 - T_1)$ & this is the heat came out from reservoir, so decrease in entropy of reservoir

$$\therefore \Delta S_{\text{reservoir}} = -\frac{\delta Q}{T_2} = -\frac{C_p(T_2 - T_1)}{T_2}$$

Total change of entropy of the composite system

$$\Delta S = \Delta S_{\text{body}} + \Delta S_{\text{reservoir}} = C_p \left[\ln(T_2/T_1) - \frac{T_2 - T_1}{T_2} \right]$$



$$\text{for } T_2 > T_1, \ln(T_2/T_1) > \frac{T_2 - T_1}{T_2}$$

\therefore Increase in entropy of the body is always greater than decrease of entropy of reservoir & \therefore entropy of universe always increases in irreversible process.

Even when the body is at higher temperature than reservoir the situation is reversed but entropy of universe increases. So whenever heat flows across finite temperature difference (~~reversible~~ ~~irreversible~~), entropy of universe increases. For reversible process entropy remains constant.

$$S_f = S_i \text{ for reversible}$$

$$S_f > S_i \text{ for irreversible}$$

This principle of increase in entropy can be termed as 2nd law of thermodynamics. Clausius restated that 1st law is energy of the universe remains constant & 2nd law is entropy of universe tends to be the maximum.

Entropy flow & entropy production

Consider heat conduction along a copper wire that are held between two reservoirs at T_1 & T_2 ($T_1 > T_2$). If heat current or rate of flow of heat is I_Q then in unit time, hot source decreases its entropy by I_Q/T_1 , sink increases entropy by I_Q/T_2 & copper wire at steady state suffers no change in entropy. So $\Delta S_{\text{universe}} = \frac{I_Q}{T_2} - \frac{I_Q}{T_1} > 0$.

We can see it from the perspective of wire that the hot source lost entropy to the wire & there is a flow of entropy into the wire of I_Q/T_1 per unit time. The sink gained entropy so there was a flow of entropy out of the wire. I_Q/T_2 .

$$\text{Entropy production in wire } \frac{dS}{dt} = \frac{I_Q}{T_2} - \frac{I_Q}{T_1} = I_Q \frac{T_1 - T_2}{T_1 T_2}$$

$$\text{If } T_1 = T + \Delta T, T_2 = T, T_1 T_2 \approx T^2$$

$$\therefore \frac{dS}{dt} = I_Q \frac{\Delta T}{T^2} = I_Q \frac{\Delta T}{T} = I_S \frac{\Delta T}{T}$$

↓
entropy
production

- Ques. 1. A reversible engine in Carnot cycle between 100°C & 0°C does work 1200 kg-metre / cycle. Find the amount of heat drawn from source. $J = 4.18 \times 10^7$ ergs/cal.

$$\frac{Q_1}{T_1} = \frac{Q_2}{T_2} = \frac{Q_1 - Q_2}{T_1 - T_2} = \frac{W}{T_1 - T_2} \quad W = mgh$$

$$\therefore Q_1 = \frac{WT_1}{T_1 - T_2} = \frac{1200 \times 10^3 \times 980 \times 373}{100 \times 4.18 \times 10^7} = 10494 \text{ cal.}$$

2. A body of constant heat capacity C_p at temperature T_i is put in contact with reservoir at T_f . At constant pressure, body comes to equilibrium. Show that entropy change of universe is $C_p[x - \ln(1+x)]$ where $x = -\frac{T_f - T_i}{T_f}$.

Gain in entropy of the body $\Delta S_{\text{body}} = \int_{T_i}^{T_f} \frac{C_p dT}{T} = C_p \ln\left(\frac{T_f}{T_i}\right)$

Loss of entropy of reservoir $\Delta S_{\text{reservoir}} = -\frac{\delta S}{T_f} = -\frac{C_p(T_f - T_i)}{T_f}$

∴ Total change in entropy of the universe $\Delta S = C_p \ln\frac{T_f}{T_i} - C_p \frac{T_f - T_i}{T_f}$
 $= (C_p R \ln(1+x)^{-1} + C_p x) = C_p(x - \ln(1+x))$

3. According to Debye's law, the molar heat capacity at constant volume of Diamond varies with temperature as

$C_V = 3R \frac{4\pi^4}{5} \left(\frac{T}{H}\right)^3$. What is the entropy change in units of R of a diamond of 1.2 gm when it's heated at constant volume from 10 to 350K. Atomic weight of carbon is 12 gm & $H = 2230$ K.

$$\Delta S = \frac{1.2}{12} \int_{10}^{350} \frac{C_V dT}{T} = \frac{1.2}{12} 3R \frac{4\pi^4}{5} \frac{1}{H} s \int_{10}^{350} T^2 dT$$

$$= 0.03 R.$$

4. m gms of water at T_1 is isobarically & adiabatically mixed with an equal mass of water at T_2 , so that they attain a common final temperature T_c . Show that entropy change of the universe is $2mC_V \ln \frac{(T_1+T_2)}{\sqrt{T_1 T_2}}$
 What would be the loss of available energy?

T_c will attend when $mC_V(T_1 - T_c) = mC_V(T_c - T_2)$

$$\text{or } T_c = \frac{T_1 + T_2}{2}. \quad \Delta S_1 = mC_V \int_{T_1}^{T_c} \frac{dT}{T} = mC_V \ln \frac{T_c}{T_1}$$

$$\Delta S_2 = mC_V \int_{T_2}^{T_c} \frac{dT}{T} = mC_V \ln \frac{T_c}{T_2}$$

$$\Delta S_{\text{universe}} = \Delta S_1 + \Delta S_2 = mC_V \ln \frac{T_c^2}{T_1 T_2} = mC_V \ln \left(\frac{T_c}{\sqrt{T_1 T_2}}\right)^2$$

$$= 2mC_V \ln \frac{T_1 + T_2}{2\sqrt{T_1 T_2}}$$

If T_0 is the lowest attainable temperature, then unavailable amount of energy is $S_{\text{unattain}} = T_0 \Delta S_{\text{universe}} = 2mC_V T_0 \ln \frac{T_1 + T_2}{2\sqrt{T_1 T_2}}$.

- HW
- The equation of state of a new matter is $pV = AT^3$, $A = \text{constant}$. Internal energy of this matter is $U = BT^n \ln(V/V_0) + f(T)$. Using first law of thermodynamics, find B and n .
 - Suppose an engine works between two reservoirs at T_1 & T_2 ($T_2 > T_1$) until both reservoirs attain final temperature T_c . Show that $T_c > \sqrt{T_1 T_2}$. What is the maximum amount of work obtainable from this engine?

Entropy & Unavailable energy

Suppose that Q amount of heat is extracted from a reservoir at temperature T_1 & it is desired to convert this heat into work as much as possible. If T_0 is the temperature of sink, then using a reversible engine, $W_{\max}^r = Q(1 - \frac{T_0}{T_1})$.

Consider an irreversible conduction of heat Q along a metal bar under finite temperature gradient from a region at temperature T_1 to a region at temperature T_2 . After conduction has taken place, Q amount of heat is available at lower temperature T_2 or the maximum amount of available work is $W_{\max}^{irr} = Q(1 - \frac{T_0}{T_2})$.

As $T_1 > T_2$, so maximum work obtained in W_{\max}^{irr} is less than W_{\max}^r .

\therefore The amount of energy that is unavailable for work is,

$$E = Q(1 - \frac{T_0}{T_1}) - Q(1 - \frac{T_0}{T_2}) = T_0 Q (\frac{1}{T_2} - \frac{1}{T_1}) = T_0 \Delta S$$

where ΔS is the entropy change of the universe due to the irreversible process. Since irreversible process is continually happening in nature, so energy is continually becoming unavailable for work. This is known as "Principle of degradation of energy".

Thus according to this principle, we are always losing available energy & then a day will come when we will have no available energy for work. This is called "heat death" of the universe.

Disorder & entropy

Whenever work is dissipated within the system because of friction, viscosity, inelasticity, electric resistance or magnetoo hysteresis, the molecules are more disordered. We learned that irreversible process is associated with an increase in entropy of the universe. Thus, entropy must be related with the disorder.

Disorder of a system depends on the number of microstates accessible to the system. This is called "Thermodynamic probability" (Ω) & we can write $S = f(\Omega)$. To obtain the functional form, consider two systems having thermodynamic probability Ω_1 and Ω_2 with entropies $S_1 = f(\Omega_1)$ and $S_2 = f(\Omega_2)$ respectively.

If these systems are mixed then the total entropy is $S = S_1 + S_2$ and the total number of microstates of the system will be

$$\Omega = \Omega_1 \Omega_2. \quad \therefore S_1 + S_2 = f(\Omega_1 \Omega_2)$$

$$\therefore f(\Omega_1) + f(\Omega_2) = f(\Omega_1 \Omega_2)$$

$$\begin{aligned} \therefore \frac{\partial f(\Omega_1)}{\partial \Omega_1} + 0 &= \frac{\partial f(\Omega_1 \Omega_2)}{\partial \Omega_1} = \frac{\partial f(\Omega_1 \Omega_2)}{\partial (\Omega_1 \Omega_2)} \frac{\partial (\Omega_1 \Omega_2)}{\partial \Omega_1} \\ &= \Omega_2 \frac{\partial f(\Omega_1 \Omega_2)}{\partial (\Omega_1 \Omega_2)} \end{aligned}$$

$$\text{Multiplying with } \Omega_1, \quad \Omega_1 \frac{\partial f(\Omega_1)}{\partial \Omega_1} = \Omega_1 \Omega_2 \frac{\partial f(\Omega_1 \Omega_2)}{\partial (\Omega_1 \Omega_2)}$$

$$\text{Similarly, } 0 + \frac{\partial f(\Omega_2)}{\partial \Omega_2} = \frac{\partial f(\Omega_1 \Omega_2)}{\partial \Omega_2} \Rightarrow \Omega_2 \frac{\partial f(\Omega_2)}{\partial \Omega_2} = \Omega_1 \Omega_2 \frac{\partial f(\Omega_1 \Omega_2)}{\partial (\Omega_1 \Omega_2)}$$

$$\therefore \Omega_1 \frac{df(\Omega_1)}{d\Omega_1} = \Omega_2 \frac{df(\Omega_2)}{d\Omega_2} \quad \therefore \Omega \frac{df(\Omega)}{d\Omega} = \text{constant (K say)}$$

$$\therefore df(\Omega) = K \frac{d\Omega}{\Omega} \quad \therefore f(\Omega) = K \ln \Omega + C$$

But if $\Omega = 1$, $f(\Omega) = 0$ as the system is in perfect ordered state.

$$\therefore C = 0. \quad S = K \ln \Omega$$

Entropy and Information

Since the disorder of a system is related to the available information, therefore the entropy of a system should also have a relation with information.

Suppose we are called upon to guess a person's first name. The number of choices of names of men & women is huge. So there is a great disorder without any information. But given an information that the person is a man, the number of choices of names is reduced. Information is reduced further that man is a physicist and this reduces the disorder. The disorder in choosing names is further reduced if given an information that the physicist is a Nobel laureate. \therefore Fewer the number of ways a particular state of a system is achieved, the greater is the information.

A measure of the information when the number of choices is reduced from Ω_0 to Ω_1 is $I = K \ln \frac{\Omega_0}{\Omega_1} = S_0 - S_1$. The bigger the reduction, the bigger the information. Also, $S_1 = S_0 - I$.

\therefore Entropy measures the lack of information about the exact state of a system.

Entropy & 2nd law of Thermodynamics

The 2nd law provides proper direction to all natural processes while the Kelvin-Planck statement gives a direction for the conversion between

heat and work, Clausius statement gives a direction for heat flow between two bodies at different temperature. Combining the 2nd law can be defined in terms of entropy as

"A natural irreversible process will always proceed in a direction in which the entropy of universe increases. In a reversible process, entropy does not change."

This means in approaching equilibrium, the entropy of the universe must increase. $S_f - S_i = \int_i^f \frac{dq}{T}$. The principle of unavailable energy is a replacement of Kelvin-Planck's statement of 2nd law since both suggest a definite rejection of heat energy if some work is to be obtained from heat. The concept of entropy with reference to disorder shows that the direction of all natural processes are governed by probability laws. This 2nd law is a statistical law which is defined as the probability of a process to occur is more if the direction of the process gives an increment in entropy of the universe.

Practical Engines

We learned, engine is a device that converts heat into work. Practical engines are of two types. (a) Internal-combustion engine (Gasoline & Diesel engine)

(b) External-combustion engine.

(Steam & Stirling engine)

In internal combustion engine, burning of fuel & oxygen with air take place in confined combustion chamber to rise P, & T of the system. In external combustion engine, high temperature surroundings transfer heat to the combustion chamber.

Otto Cycle & Gasoline Engine

The cycle involves performance of 6 processes, 4 of which require "stroke" (vertical motion) of the piston

1. Intake stroke:

A mixture of gasoline vapour and air is drawn into the combustion chamber at constant pressure by the succession of the piston, represented by $5 \rightarrow 1$, & according to equation $P_0 V = n R T_1$, P_0 = atmospheric pressure, V = vol. of chamber, T_1 = temperature of air outside & n moles of atom.

2. Compression stroke: The mixture is compressed in quasi-static adiabatic stroke until its pressure & temperature rise considerably. No friction & no loss of heat at the cylinder wall. $1 \rightarrow 2$ process is represented as. $T_1 V_1^{\gamma-1} = T_2 V_2^{\gamma-1}$

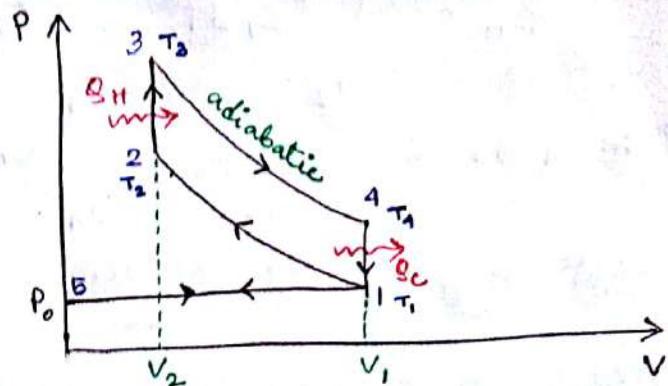
3. Ignition or combustion: Process $2 \rightarrow 3$ represents quasi-static isochoric increase of temperature from T_2 to T_3 where combustion takes place very rapidly by an electric spark.

4. Power stroke: The hot combustion product expands and push the piston out. $3 \rightarrow 1$ represents quasi-static adiabatic process so that temperature falls down to T_4 with equation $T_3 V_2^{\gamma-1} = T_4 V_1^{\gamma-1}$

5. Valve Exhaust: The combusted product at the end of power stroke are still at a higher pressure and temperature than outside. An exhaust valve allows the gas mixture to escape, that is $1 \rightarrow 5$.

6. Exhaust Stroke: The piston pushes all the remaining combustion product out of the chamber by exerting a sufficiently large pressure $1 \rightarrow 5$ represents this isobaric exhaust.

If the specific heat C_V of the gas mixture is constant then heat absorbed in process $2 \rightarrow 3$ is



$Q_H = \int_{T_2}^{T_3} C_V dT = C_V(T_3 - T_2)$. and the heat rejected in $1 \rightarrow 1$ is $Q_C = - \int_{T_1}^{T_4} C_V dT = C_V(T_4 - T_1)$. Thus the thermal efficiency of the gasoline engine is $\eta = 1 - \frac{Q_C}{Q_H} = 1 - \frac{T_4 - T_1}{T_3 - T_2}$

Also for the two adiabatic process we know

$$T_1 V_1^{\gamma-1} = T_2 V_2^{\gamma-1}, \quad T_4 V_1^{\gamma-1} = T_3 V_2^{\gamma-1}$$

$$\therefore (T_4 - T_1) V_1^{\gamma-1} = (T_3 - T_2) V_2^{\gamma-1} \Rightarrow \frac{T_4 - T_1}{T_3 - T_2} = \left(\frac{V_2}{V_1}\right)^{\gamma-1}$$

$$\therefore \eta = 1 - \left(\frac{1}{r}\right)^{\gamma-1} \quad \text{where } r = \frac{V_1}{V_2} \text{ is the compression ratio.}$$

In actual gasoline engine $r \leq 10$. If r is large then temperature of the gasoline-air mixture upon compression is huge enough to cause combustion before the advent of spark. This is called "Preignition." Taking $r=9$, $\gamma=1.4$, we obtain

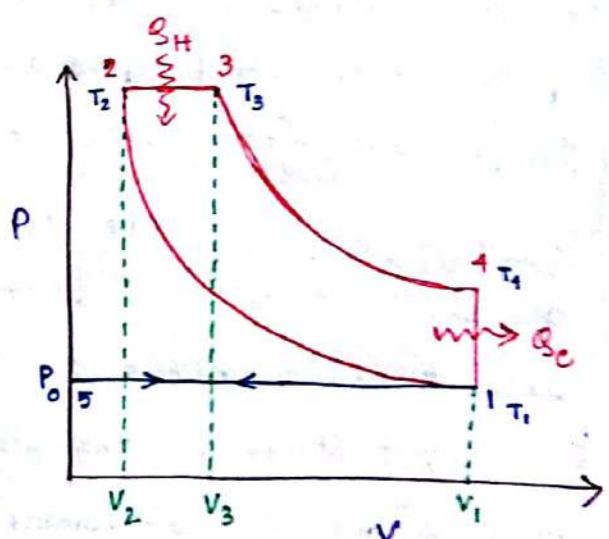
$\eta = \underline{\underline{67\%}}$. In actual engine however due to turbulence, heat conduction etc, the efficiency is less than 67%.

Diesel engine

Rudolf Diesel in 1897 designed this air-standard Diesel cycle where only air is admitted on intake stroke. The air is compressed adiabatically until temperature reaches high to ignite oil after compression stroke. Combustion stroke happens isobarically & piston moves out. Process 4,5,6

happen like the Otto cycle. Only change is $2 \rightarrow 3$ is horizontal.

$$\text{Quasistatic isobaric heat absorption } (2 \rightarrow 3) \quad Q_H = \int_{T_2}^{T_3} C_p dT = C_p(T_3 - T_2)$$



and valve exhaust releases (4→1) Q_c heat as in Otto engine,

$$Q_c = - \int_{T_4}^{T_1} C_V dT = C_V (T_1 - T_4). \text{ Thus the thermal efficiency of an idealized diesel engine is } \eta = 1 - \frac{Q_c}{Q_H} = 1 - \frac{T_1 - T_4}{\gamma(T_2 - T_1)}.$$

We want to calculate the ratio in terms of adiabatic expansion ratio $\rho = \frac{V_1}{V_2}$ and combustion expansion ratio $e = \frac{V_3}{V_2}$. So we've to evaluate $\frac{T_1 - T_4}{T_3 - T_2}$ in that.

for path 1→2, adiabatic

$$T_1 V_1^{\gamma-1} = T_2 V_2^{\gamma-1}$$

$$\Rightarrow T_1 = T_2 \left(\frac{V_2}{V_1} \right)^{\gamma-1} = T_2 \frac{1}{\rho^{\gamma-1}}$$

for path 2→3 isobaric

$$\frac{T_3}{V_3} = \frac{T_2}{V_2}$$

$$\Rightarrow T_3 = T_2 \frac{V_3}{V_2} = T_2 e$$

for path 3→4, adiabatic

$$T_3 V_3^{\gamma-1} = T_4 V_4^{\gamma-1}, \text{ as } V_1 = V_4$$

$$\Rightarrow T_4 = T_3 \left(\frac{V_3}{V_4} \right)^{\gamma-1} = T_3 \left(\frac{V_3}{V_2} \right)^{\gamma-1} \left(\frac{V_2}{V_1} \right)^{\gamma-1}$$

$$= T_3 e^{\frac{\gamma-1}{\rho^{\gamma-1}}} = T_2 e^{\frac{1}{\rho^{\gamma-1}}}$$

$$\therefore \eta = 1 - \frac{1}{\gamma} \left[\frac{T_2 e^{\frac{1}{\rho^{\gamma-1}}} - T_2 \frac{1}{\rho^{\gamma-1}}}{T_2 e - T_2} \right] = 1 - \frac{1}{\gamma} \frac{1}{\rho^{\gamma-1}} \left[\frac{e^{\frac{1}{\rho^{\gamma-1}}} - 1}{e - 1} \right]$$

$$\text{if } e = 5, \gamma = 1.4, T_1 = 300K, T_2 = 990K, \eta = 59\% \quad \underline{\underline{=}}$$

Efficiency of Diesel engine < Efficiency of Otto engine

Q1 1. find the efficiency of a Carnot's engine working between the steam point and the ice point.

$$T_1 = 100^\circ C = 373K, T_2 = 0^\circ C = 273K.$$

$$\eta = 1 - \frac{T_2}{T_1} = 1 - \frac{273}{373} = \frac{100}{373}. \quad \therefore \% \text{ efficiency} = \frac{100}{373} \times 100 \\ = 26.81\%$$

2. find the efficiency of a Carnot's engine working between $127^\circ C$ and $27^\circ C$. It absorbs 80 cals of heat. How much heat is rejected?

$$T = 400\text{K}, T_2 = 300\text{K}, \eta = 1 - \frac{T_2}{T_1} = 1 - \frac{300}{400} = 0.25$$

$$\therefore \eta = 25\%, W = \eta Q_1 = 0.25 \times 80 = 20 \text{ cals.}$$

$$\text{Heat rejected } Q_2 = Q_1 - W = 80 - 20 = 60 \text{ cals.}$$

HW 1. A Carnot engine has an efficiency of 30% when the sink temperature is 27°C . What must be the change in temperature of the source to make its efficiency 50%?

2. An inventor claims to have developed an engine working between 600K and 300K to deliver an efficiency of 52%. Is this claim valid? [Hint: use Carnot's theorem].

3. Two Carnot engines X & Y are operating in series. X receives heat at 1200K & rejects to a reservoir at temperature $T\text{K}$. The 2nd engine Y receives the heat rejected by X & in turn rejects to a heat reservoir at 300K . Calculate the temperature T for the situation when, reservoir at 300K . Calculate the temperature T for the situation when,

- (i) The work outputs of two engines are equal.
(ii) The efficiency of two engines are equal.

4. A Carnot's refrigerator takes heat from water at 0°C and discards it to a room temperature at 27°C . 1kg of water at 0°C is to be changed into ice at 0°C . How many calories of heat are discarded to the room? What is the work done by the refrigerator in this process? What is the coefficient of performance ($P = \frac{Q_2}{Q_1 - Q_2}$) of the machine? [1 Cal = 4.2 Joule].

Third law of Thermodynamics : Nernst's Heat Theorem

In 1906, Nernst proposed a general principle on atomic heat of solids at low temperature. The third law of thermodynamics states that the heat capacities of all solids tend to zero as the absolute zero of temperature is approached & the internal energies and entropies of all substances become equal there, approaching their common value

Asymptotically tending to zero. In terms of entropy, at absolute zero temperature, the entropy tends to zero & the molecules of a substance are in perfect order.

We cannot have less than zero K temperature because if $T_2 < 0\text{K}$ then efficiency $\eta = 1 - \frac{T_2}{T_1} = 1 + \frac{T_2}{T_1} > 1$. So we cannot make more than 100% efficient engine.

Maxwell's Thermodynamical Relationship

from the 1st law of thermodynamics, Maxwell derived 6 fundamental thermodynamic relations using P, V, T, S with any two pair as dependent & other as independent variable.

$$\text{from 1st law, } dQ = dU + PdV \quad \& \text{ from 2nd law, } dS = Tds \\ \therefore dU = dQ - PdV = Tds - PdV \quad \text{--- (1)}$$

Here dU, dS, dV are all exact differentials. Considering U, S, V as function of two independent variables x & y where $\{x, y\}$ = combination of P, V, T, S any two. Then,

$$dU = \left(\frac{\partial U}{\partial x}\right)_y dx + \left(\frac{\partial U}{\partial y}\right)_x dy, \quad dS = \left(\frac{\partial S}{\partial x}\right)_y dx + \left(\frac{\partial S}{\partial y}\right)_x dy$$

$$dV = \left(\frac{\partial V}{\partial x}\right)_y dx + \left(\frac{\partial V}{\partial y}\right)_x dy$$

$$\text{Substituting in (1), } \left(\frac{\partial U}{\partial x}\right)_y dx + \left(\frac{\partial U}{\partial y}\right)_x dy = T \left[\left(\frac{\partial S}{\partial x}\right)_y dx + \left(\frac{\partial S}{\partial y}\right)_x dy \right] - P \left[\left(\frac{\partial V}{\partial x}\right)_y dx + \left(\frac{\partial V}{\partial y}\right)_x dy \right]$$

$$= \left[T \left(\frac{\partial S}{\partial x}\right)_y - P \left(\frac{\partial V}{\partial x}\right)_y \right] dx + \left[T \left(\frac{\partial S}{\partial y}\right)_x - P \left(\frac{\partial V}{\partial y}\right)_x \right] dy$$

Comparing the coefficients of dx & dy, we get

$$\left(\frac{\partial U}{\partial x}\right)_y = T \left(\frac{\partial S}{\partial x}\right)_y - P \left(\frac{\partial V}{\partial x}\right)_y, \quad \left(\frac{\partial U}{\partial y}\right)_x = T \left(\frac{\partial S}{\partial y}\right)_x - P \left(\frac{\partial V}{\partial y}\right)_x$$