Interconnection of the Kirchhoff plate within the port-Hamiltonian framework

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- 1 Introduction: discretization of pH Systems
- The Kirchhoff plate as a port-Hamiltonian system
 - Classical formulation
 - Port-Hamiltonian formulation
- 3 Structure preserving discretization
 - The partitioned finite element method
 - Application to the Kirchhoff plate
- 4 Interconnection with rigid elements
- 5 Stabilization by boundary injection

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Infinite dimensional pH systems

General infinite dimensional pH system

$$\begin{cases} \frac{\partial x}{\partial t}(z,t) &= \mathcal{J}\frac{\delta H}{\delta x} + B\mathbf{u}(z,t), \\ \mathbf{y}(z,t) &= B^*\frac{\delta H}{\delta x}. \end{cases}$$

With boundary conditions

$$u_{\partial} = \mathcal{B} \frac{\delta H}{\delta x}, \quad y_{\partial} = \mathcal{C} \frac{\delta H}{\delta x}$$

Energy rate:
$$\dot{H} = \int_{\partial\Omega} u_{\partial}y_{\partial} \; \mathrm{d}s + \int_{\Omega} u(z,t)y(z,t) \; \mathrm{d}\Omega$$

- lacksquare x energy variables, $e=\delta_x H=:$ co-energy variables ;
- ullet \mathcal{J} : skew-symmetric differential operator;
- \blacksquare \mathcal{B}, \mathcal{C} : boundary operator;
- \bullet u, y, B: distributed input, output and control operator.

Infinite dimensional pH systems

Linear infinite dimensional pH system

$$\begin{cases} \frac{\partial x}{\partial t}(z,t) &= \mathcal{J}\mathcal{Q}x + Bu(z,t), \\ y(z,t) &= B^*\mathcal{Q}x. \end{cases}$$

With boundary conditions

$$u_{\partial} = \mathcal{B}\mathcal{Q}x, \quad y_{\partial} = \mathcal{C}\mathcal{Q}x$$

Energy rate:
$$\dot{H} = \int_{\partial\Omega} u_{\partial} y_{\partial} \; \mathrm{d}s + \int_{\Omega} \frac{u(z,t)y(z,t)}{\mathrm{d}\Omega} \; \mathrm{d}\Omega$$

- x energy variables, $e = \delta_x H = \mathcal{Q}x$: co-energy variables (\mathcal{Q} symmetric positive operator);
- ullet \mathcal{J} : skew-symmetric differential operator;
- \blacksquare \mathcal{B}, \mathcal{C} : boundary operator;
- u, y, B: distributed input, output and control operator.

Structure preserving discretization

The main properties of the dpH (conservative, passive system) are preserved at a discrete level.

Infinite dimensional pHs

PDE:

$$\partial_t x(z,t) = \mathcal{J} \frac{\delta H}{\delta x} + B u(z,t),$$
 $y(z,t) = B^* \frac{\delta H}{\delta x}.$

Boundary conditions:

$$u_{\partial} = \mathcal{B} \frac{\delta H}{\delta x}, \quad y_{\partial} = \mathcal{C} \frac{\delta H}{\delta x}$$

Power balance (Stokes Theorem):

$$\dot{H} = \int_{\partial\Omega} u_{\partial}y_{\partial} \, \mathrm{d}s + \int_{\Omega} u(z,t)y(z,t) \, \mathrm{d}\Omega$$

Finite dimensional pHs

ODE:

$$\dot{x} = J\partial_x H + B_d u_d + B_\partial u_\partial,$$

$$y_d = B_d^T \partial_x H,$$

$$y_\partial = B_\partial^T \partial_x H$$

Power balance:

$$\dot{H} = u_{\partial}^T y_{\partial} + u_{d}^T y_{d}$$

Structure preserving discretization

The main properties of the dpH (conservative, passive system) are preserved at a discrete level.

Infinite dimensional pHs (linear case)

PDF:

$$\partial_t x(z,t) = \mathcal{J} \mathcal{Q} x + B u(z,t),$$

 $y(z,t) = B^* \mathcal{Q} x.$

Boundary conditions:

$$u_{\partial} = \mathcal{B}\mathcal{Q}x, \quad y_{\partial} = \mathcal{C}\mathcal{Q}x$$

Power balance (Stokes Theorem):

$$\dot{H} = \int_{\partial\Omega} u_{\partial}y_{\partial} \, \mathrm{d}s + \int_{\Omega} u(z,t)y(z,t) \, \mathrm{d}\Omega$$

Finite dimensional pHs (linear case)

ODE:

$$\dot{x} = JQx + B_d u_d + B_\partial u_\partial,$$

$$y_d = B_d^T Q x,$$

$$y_\partial = B_\partial^T Q x$$

Power balance:

$$\dot{H} = u_{\partial}^T y_{\partial} + u_{d}^T y_{d}$$

Introduction: discretization of pH Systems

Available methods

- Spectral methods (Moulla 2012):
 - √ Rapid spectral convergence;
 - × Only for 1D problem;
- Finite differences (Trenchant 2018);
 - √ Valid up to 2D geometries;
 - × Requires ad hoc implementation (staggered grids);
- Finite elements based
 - Golo 2004, Kotyczka 2018: the implementation requires exterior calculus knowledge and depends on the some parameters that ensure the preservation of power flow:
 - Cardoso-Ribeiro 2018:
 - ✓ Natural extension of the mixed finite element method to pH systems;
 - ✓ Implementable using well-established libraries (Fenics, Firedrake);

Why structure preserving discretization?

A distributed pH system interconnected to a strictly positive real boundary controller is asymptotically stable¹.

This allows the construction of observed based controller based on a finite dimensional discretization of the plant² that guarantees stability of the infinite dimensional system (no spillover effect).

¹J. A. Villegas et al. "Exponential Stability of a Class of Boundary Control Systems". In: *IEEE Transactions on Automatic Control* 54.1 (2009), pp. 142–147. ISSN: 2334–3303. DOI: 10.1109/TAC.2008.2007176.

²Jesus Toledo et al. *Observer-based boundary control of distributed port-Hamiltonian systems*. 2019. arXiv: 1910.04193.

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The classical Kirchhoff model

Classical bilaplacian formulation

For an homogeneous isotropic material

$$\rho h \frac{\partial^2 w}{\partial t^2} + D\Delta^2 w = p, \quad \boldsymbol{x} \in \Omega \subset \mathbb{R}^2.$$

 $\Delta^2 = \partial_{xxx} + 2\partial_{xxyy} + \partial_{yyyy}$ is the bilaplacian operator

- ρ [kg/m³] is the mass density;
- \bullet h [m] is the plate thickness;
- $p [N/m^2]$ is an external distributed force;
- $lue{D}$ [Pa m] is the bending stiffness;

The classical Kirchhoff model

Bending moment formulation

$$\rho h \frac{\partial^2 w}{\partial t^2} + \operatorname{div} \operatorname{Div}(\boldsymbol{M}) = p, \quad \boldsymbol{x} \in \Omega \subset \mathbb{R}^2.$$

Where $M = \mathbb{D}\nabla^2 w \in \mathbb{R}^{2\times 2}_{\text{sym}}$ is the bending moment tensor and $\nabla^2 = \operatorname{Grad} \circ \operatorname{grad}$ the Hessian.

$$\operatorname{div}\operatorname{Div}(\boldsymbol{M}) = \partial_{xx}M_{11} + 2\partial_{xy}M_{12} + \partial_{yy}M_{22}$$

- ρ [kg/m³] is the mass density;
- *h* [m] is the plate thickness:
- p [N/m²] is an external distributed force:
- D is the bending rigidity tensor (symmetric, positive). For an homogeneous isotropic material

$$\mathbb{D}\boldsymbol{A} = D\left\{ (1 - \nu)\boldsymbol{A} + \nu \operatorname{Tr}(\boldsymbol{A})\boldsymbol{I} \right\};$$

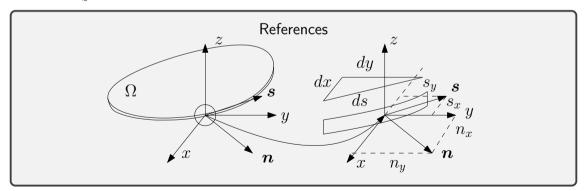
Boundary conditions

For the boundary variables consider the definitions

Flexural moment $M_{nn} = M : (n \otimes n),$ Shear stress $q = \operatorname{Div} M,$

Torsional moment $M_{ns}={m M}$: $({m n}\otimes{m s}),$ Kirchhoff shear force $\widetilde{q}_n=-{m q}\cdot{m n}-\partial_{m s}M_{ns}.$

 $m{A}: m{B} = \sum_{i,j} A_{ij} B_{ij}$ is the tensor contraction and $m{a} \otimes m{b} = m{a} m{b}^{ op} \in \mathbb{R}^{2 imes 2}$ is the dyadic product.



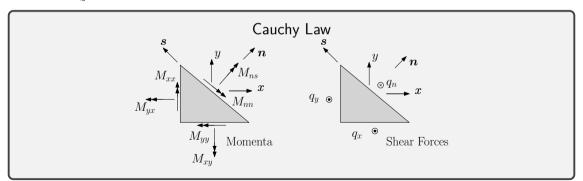
Boundary conditions

For the boundary variables consider the definitions

Flexural moment $M_{nn} = M : (n \otimes n)$, Shear stress q = Div M,

Torsional moment $M_{ns} = M : (n \otimes s)$, Kirchhoff shear force $\widetilde{q}_n = -q \cdot n - \partial_s M_{ns}$.

 $m{A}: m{B} = \sum_{i,j} A_{ij} B_{ij}$ is the tensor contraction and $m{a} \otimes m{b} = m{a} m{b}^{ op} \in \mathbb{R}^{2 \times 2}$ is the dyadic product.



Boundary conditions

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 $m{A}: m{B} = \sum_{i,j} A_{ij} B_{ij}$ is the tensor contraction and $m{a} \otimes m{b} = m{a} m{b}^ op \in \mathbb{R}^{2 imes 2}$ is the dyadic product.

Boundary conditions

$\Gamma_f = \{\widetilde{q}_n, \ M_{nn} \text{ known}\}$ $\Gamma_c = \{w, \ \frac{\partial w}{\partial n} \text{ known}\}$ Ω $\Gamma_s = \{w, \ M_{nn} \text{ known}\}$

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Hamiltonian and energy variables

The total energy of the system is given by the sum of kinetic and deformation energy

$$H = \frac{1}{2} \int_{\Omega} \left\{ \rho h \left(\partial_t w \right)^2 + \mathbb{D} \nabla^2 w : \nabla^2 w \right\} d\Omega$$

Consider the following choice for the energy variables

$$\alpha_1 := \rho h \partial_t w, \quad \text{Linear momentum}$$

$$oldsymbol{A}_2 :=
abla^2 w, \quad \mathsf{Curvature}$$

This leads to the following co-energy variables

$$e_1 := \frac{\delta H}{\delta \alpha_1} = \partial_t w$$
, Vertical velocity

$$oldsymbol{E}_2 := rac{\delta H}{\delta oldsymbol{A}_2} = oldsymbol{M}, \quad ext{Bending moment}$$

Port Hamiltonian formulation

The system

$$\frac{\partial}{\partial t} \begin{pmatrix} \alpha_1 \\ \mathbf{A}_2 \end{pmatrix} = \underbrace{\begin{bmatrix} 0 & -\operatorname{div}\operatorname{Div} \\ \nabla^2 & 0 \end{bmatrix}}_{\mathcal{J}} \begin{pmatrix} e_1 \\ \mathbf{E}_2 \end{pmatrix}, \quad \begin{pmatrix} e_1 \\ \mathbf{E}_2 \end{pmatrix} = \underbrace{\begin{bmatrix} (\rho h)^{-1} & 0 \\ 0 & \mathbb{D} \end{bmatrix}}_{\mathcal{Q}} \begin{pmatrix} \alpha_1 \\ \mathbf{A}_2 \end{pmatrix}$$

defines a Stokes-Dirac structure. The proof is readily obtained considering that the following holds

$$(\operatorname{div}\operatorname{Div})^* = \nabla^2$$

This means that the operator $\mathcal J$ is formally skew-adjoint.

It is worth noticing that the boundary variables are defined by the power balance

$$\dot{H} = \int_{\partial \Omega} \left\{ \partial_t w \, \widetilde{q}_n + \partial_n (\partial_t w) \, M_{nn} \right\} \, \mathrm{d}s.$$

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The partitioned finite element method (PFEM)

General form of a linear pH system in co-energy variables

$$\mathcal{M}\frac{\partial e}{\partial t} = \mathcal{J}e, \qquad \mathcal{M} = \mathcal{Q}^{-1}$$

General procedure for PFEM

1 Put the system into weak form:

$$\left(v, \mathcal{M} \frac{\partial e}{\partial t}\right)_{\Omega} = (v, \mathcal{J}e)_{\Omega}.$$

Apply integration by part on a partition of \mathcal{J} :

$$(v, \mathcal{J}e)_{\Omega} = j(v, e)_{\Omega} + b(v, u_{\partial})_{\partial\Omega},$$

so that $j(v,e)_{\Omega}$ is a skew-symmetric bilinear form.

Discretization by Galerkin method (same basis function for test and co-energy variables)

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Consider the Kirchhoff plate operators

$$\mathcal{M} = \mathsf{Diag}(
ho h, \; \mathbb{D}^{-1}), \quad \mathcal{J} = egin{pmatrix} 0 & -\operatorname{div} \operatorname{Div} \
abla^2 & 0 \end{pmatrix}$$

Consider the Kirchhoff plate operators

$$\mathcal{M} = \mathsf{Diag}(
ho h, \; \mathbb{D}^{-1}), \quad \mathcal{J} = egin{pmatrix} 0 & -\operatorname{div} \operatorname{Div} \
abla^2 & 0 \end{pmatrix}$$

Either the first line of the operator \mathcal{J} is integrated by parts

$$(v, \mathcal{J}e)_{\Omega} = \int_{\Omega} \left\{ -v_1 \operatorname{div} \operatorname{Div} \mathbf{E}_2 + \mathbf{V}_2 : \nabla^2 e_1 \right\} d\Omega$$

$$= \underbrace{\int_{\Omega} \left\{ -\nabla^2 v_1 : \mathbf{E}_2 + \mathbf{V}_2 : \nabla^2 e_1 \right\} d\Omega}_{j_{\mathsf{Hess}}(v,e)} + \underbrace{\int_{\partial\Omega} \left\{ v_1 \mathbf{q}_n + \partial_n v_1 \mathbf{M}_{nn} \right\} ds}_{b_N(v,\mathbf{u}_{\partial})_{\partial\Omega}}$$

Consider the Kirchhoff plate operators

$$\mathcal{M} = \mathsf{Diag}(\rho h, \; \mathbb{D}^{-1}), \quad \mathcal{J} = \begin{pmatrix} 0 & -\operatorname{div}\operatorname{Div} \\
abla^2 & 0 \end{pmatrix}$$

Either the second line of the operator \mathcal{J} is integrated by parts

$$(v, \mathcal{J}e)_{\Omega} = \int_{\Omega} \left\{ -v_1 \operatorname{div} \operatorname{Div} \mathbf{E}_2 + \mathbf{V}_2 : \nabla^2 e_1 \right\} d\Omega$$

$$= \underbrace{\int_{\Omega} \left\{ -v_1 \operatorname{div} \operatorname{Div} \mathbf{E}_2 + \operatorname{div} \operatorname{Div} \mathbf{V}_2 e_1 \right\} d\Omega}_{j_{\operatorname{div} \operatorname{Div}}(\mathbf{v}, \mathbf{e})} + \underbrace{\int_{\partial \Omega} \left\{ v_{q_n} \partial_t w + v_{m_n} \partial_n \partial_t w \right\} ds}_{b_D(\mathbf{v}, \mathbf{u}_{\partial})_{\partial \Omega}},$$

where $v_{q_n} = -(\operatorname{Div} \mathbf{V}_2) \cdot \mathbf{n} - \partial_{\mathbf{s}} v_{m_s}, \ v_{m_s} = \mathbf{V}_2 : (\mathbf{n} \otimes \mathbf{s}), \ v_{m_n} = \mathbf{V}_2 : (\mathbf{n} \otimes \mathbf{n})$

Consider the Kirchhoff plate operators

$$\mathcal{M} = \mathsf{Diag}(\rho h, \; \mathbb{D}^{-1}), \quad \mathcal{J} = \begin{pmatrix} 0 & -\operatorname{div}\operatorname{Div} \\ \nabla^2 & 0 \end{pmatrix}$$

The selection depends on the control variables. For Neumann control the first line is integrated by parts. For Dirichlet control the second.

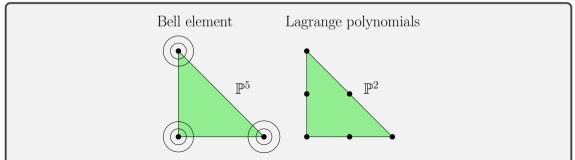
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Finite element choice

Selecting as control variables the forces and torques (Neumann boundary conditions), the following weak form is obtained:

$$m(\boldsymbol{v}, \partial_t \boldsymbol{e}) = j_{\mathsf{Hess}}(\boldsymbol{v}, \boldsymbol{e}) + b_N(\boldsymbol{v}, \boldsymbol{u}_{\partial})_{\partial\Omega}.$$

For both e_1, E_2 the H^2 conforming Bell elements are selected. Dirichlet boundary conditions are enforced by Lagrange multipliers, i.e. $u_{\partial}|_{\Gamma_D} = \lambda$. Those are discretized using quadratic Lagrange polynomials.



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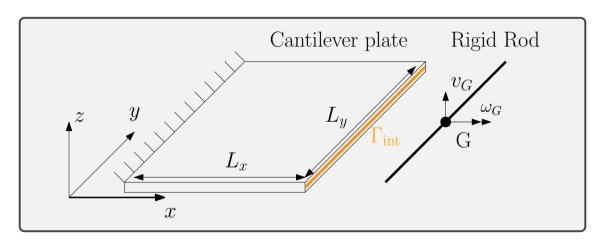
$$m(\boldsymbol{v}, \partial_t \boldsymbol{e}) = j_{\mathsf{Hess}}(\boldsymbol{v}, \boldsymbol{e}) + b_N(\boldsymbol{v}, \boldsymbol{u}_{\partial})_{\partial\Omega}.$$

For both e_1, E_2 the H^2 conforming Bell elements are selected. Dirichlet boundary conditions are enforced by Lagrange multipliers, i.e. $u_{\partial}|_{\Gamma_D} = \lambda$. Those are discretized using quadratic Lagrange polynomials.

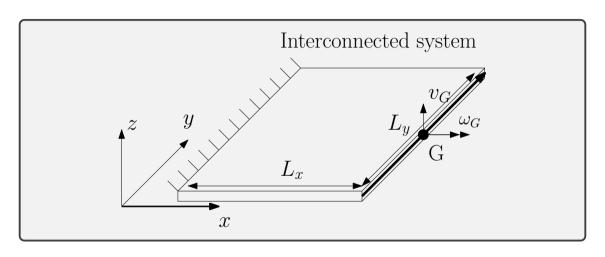
$$\begin{bmatrix} M & 0 \\ 0 & 0 \end{bmatrix} \frac{d}{dt} \begin{pmatrix} \mathbf{e} \\ \mathbf{\lambda} \end{pmatrix} = \begin{bmatrix} J & G \\ -G^T & 0 \end{bmatrix} \begin{pmatrix} \mathbf{e} \\ \mathbf{\lambda} \end{pmatrix} + \begin{bmatrix} B \\ 0 \end{bmatrix} \mathbf{u},$$
$$y = \begin{bmatrix} B^T & 0 \end{bmatrix} \begin{pmatrix} \mathbf{e} \\ \mathbf{\lambda} \end{pmatrix},$$

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Rigid rod welded to a cantilever plate



Rigid rod welded to a cantilever plate



Boundary interconnection of the Kirchhoff plate

The system is composed by a cantilever plate (distributed pH) connected to a rigid rod

$$\mathsf{dpH} \begin{cases} \frac{\partial x_1}{\partial t} = \mathcal{J} \frac{\delta H_1}{\delta x_1} \\ u_{\partial,1} = \mathcal{B} \frac{\delta H_1}{\delta x_1} \\ y_{\partial,1} = \mathcal{C} \frac{\delta H_1}{\delta x_1} \end{cases} \qquad \mathsf{pH} \begin{cases} \frac{d \boldsymbol{x}_2}{dt} = J \frac{\partial H_2}{\partial \boldsymbol{x}_2} + B \boldsymbol{u}_2 \\ \boldsymbol{y}_2 = B^T \frac{\partial H_2}{\partial x_2} + D \boldsymbol{u}_2 \end{cases},$$

where $u_{\partial,1} \in \mathcal{U}$, $y_{\partial,1} \in \mathcal{Y} = \mathcal{U}'$ belong to some Hilbert spaces and $x_2 \in \mathbb{R}^n$, $u,y \in \mathbb{R}^m$. The interconnection is power-preserving if

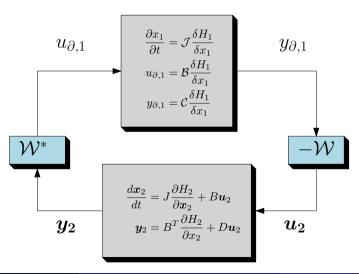
$$\langle u_{\partial,1}, y_{\partial,1} \rangle_{\mathscr{U} \times \mathscr{U}} + \langle u_2, y_2 \rangle_{\mathbb{R}^m} = 0.$$

This is achieved by introducing a compact operator $\mathcal{W}: \mathscr{Y} \to \mathbb{R}^m$

$$u_2 = -\mathcal{W} y_{\partial,1}, \qquad u_{\partial,1} = \mathcal{W}^* y_2,$$

Boundary interconnection of the Kirchhoff plate

The system is composed by a cantilever plate (distributed pH) connected to a rigid rod



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Boundary interconnection of the Kirchhoff plate

$$\begin{aligned} & \text{Plate } (\Omega = [0, L_x] \times [0, L_y]) & \text{Rigid rod} \\ \begin{bmatrix} \rho h & 0 \\ 0 & \mathbb{D}^{-1} \end{bmatrix} \frac{\partial}{\partial t} \begin{bmatrix} \partial_t w \\ \boldsymbol{M} \end{bmatrix} & = \begin{bmatrix} 0 & -\operatorname{div}\operatorname{Div} \\ \nabla^2 & 0 \end{bmatrix} \begin{bmatrix} \partial_t w \\ \boldsymbol{M} \end{bmatrix} & \begin{bmatrix} M & 0 \\ 0 & J_G \end{bmatrix} \frac{d}{dt} \begin{pmatrix} v_G \\ \omega_G \end{pmatrix} = \begin{pmatrix} F_z \\ T_x \end{pmatrix} = \boldsymbol{u}_{\mathrm{rod}}, \\ & & & & & & & \\ & & & & & & \\ & & & & & & \\ & & & & & & \\ & & & & & & \\ & & & & & & \\ & & & & & & \\ & & & & & \\ & & & & & \\ & & & & & \\ & & & & & \\ & & & & & \\ & & & & & \\ & & & & & \\ & & & & & \\ & & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & & \\ & & \\ & &$$

Space \mathscr{Y} is the space of square-integrable functions with support on $\Gamma_{\text{int}}=\{(x,y)|\ x=L_x, 0\leq y\leq L_y\}$. The interconnection operator then provides the total force and torque acting on the rigid rod

$$\mathcal{W}y_{\partial,\mathsf{pl}} = -\begin{pmatrix} F_z \\ T_x \end{pmatrix} = \begin{pmatrix} \int_{\Gamma_{\mathsf{int}}} y_{\partial,\mathsf{pl}} \, \mathrm{d}s \\ \int_{\Gamma_{\mathsf{int}}} (y - L_y/2) \, y_{\partial,\mathsf{pl}} \, \mathrm{d}s \end{pmatrix}.$$

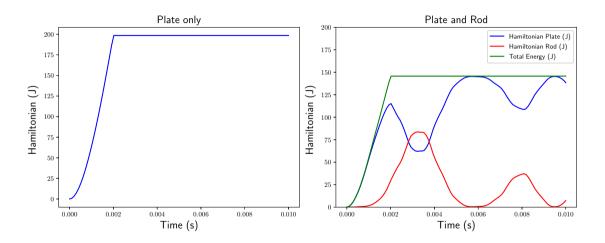
The adjoint operator provides a rigid movement as the plate input at Γ_{int}

$$\begin{split} \left\langle \mathcal{W} y_{\partial, \mathrm{pl}}, \; \boldsymbol{y}_{\mathrm{rod}} \right\rangle_{\mathbb{R}^m} &= \left\langle y_{\partial, \mathrm{pl}}, \, \mathcal{W}^* \boldsymbol{y}_{\mathrm{rod}} \right\rangle_{L^2(\Gamma_{\mathrm{int}})}, \\ \mathcal{W}^* y_{\mathrm{rod}} &= v_G + \omega_G \left(y - L_y / 2 \right). \end{split}$$

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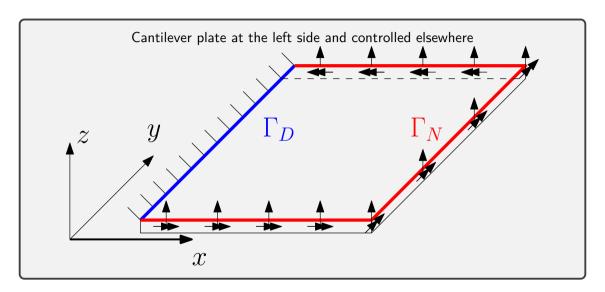
Results

$$p = \begin{cases} \text{Distributed load } (t_{\mathsf{end}} = 10 \, [\text{ms}]) \\ 10^5 \left[y + 10 \, (y - L_y/2)^2 \right] [Pa], & \forall \, t < 2 \, [\text{ms}], \\ 0, & \forall \, t \geq 2 \, [\text{ms}]. \end{cases}$$



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Boundary control of a cantilever plate



Boundary stabilization of the Kirchhoff plate

Consider the problem

$$\begin{bmatrix} \rho h & 0 \\ 0 & \mathbb{D}^{-1} \end{bmatrix} \frac{\partial}{\partial t} \begin{bmatrix} \partial_t w \\ \mathbf{M} \end{bmatrix} = \begin{bmatrix} 0 & -\operatorname{div}\operatorname{Div} \\ \nabla^2 & 0 \end{bmatrix} \begin{bmatrix} \partial_t w \\ \mathbf{M} \end{bmatrix} \quad (x, y) \in \Omega = [0, 1] \times [0, 1]$$

subjected to the following boundary conditions

$$\begin{array}{l} \partial_t w | \Gamma_D = 0, \\ \partial_x \partial_t w | \Gamma_D = 0, \end{array} \qquad \Gamma_D = \{x = 0\} \\ M_{nn} | \Gamma_N = u_M, \\ \widetilde{q} | \Gamma_N = u_F, \end{array} \qquad \Gamma_N = \{y = 0 \cup x = 1 \cup y = 1\} \end{array}$$

with initial conditions (compatible with the constraints):

$$\partial_t w(x, y, 0) = x^2;$$
 $\mathbf{M}(x, y, 0) = 0.$

Boundary stabilization of the Kirchhoff plate

Obtain a finite-dimensional uncontrolled system

$$\begin{bmatrix} M & 0 \\ 0 & 0 \end{bmatrix} \frac{d}{dt} \begin{pmatrix} \mathbf{e} \\ \mathbf{\lambda} \end{pmatrix} = \begin{bmatrix} J & G \\ -G^T & 0 \end{bmatrix} \begin{pmatrix} \mathbf{e} \\ \mathbf{\lambda} \end{pmatrix} + \begin{bmatrix} B \\ 0 \end{bmatrix} \mathbf{u},$$
$$y = \begin{bmatrix} B^T & 0 \end{bmatrix} \begin{pmatrix} \mathbf{e} \\ \mathbf{\lambda} \end{pmatrix},$$

Apply the control law $\boldsymbol{u} = -K\boldsymbol{y},\ K > 0$

$$\begin{bmatrix} M & 0 \\ 0 & 0 \end{bmatrix} \frac{d}{dt} \begin{pmatrix} \mathbf{e} \\ \boldsymbol{\lambda} \end{pmatrix} = \begin{bmatrix} J - R & G \\ -G^T & 0 \end{bmatrix} \begin{pmatrix} \mathbf{e} \\ \boldsymbol{\lambda} \end{pmatrix},$$

with $R = BKB^T \succeq 0$.

The Hamiltonian $\dot{H} = -e^T Re \le 0$ is a non increasing function and by La Salle principle the equilibrium point e = 0 is asymptotically stable.

Nice, 13/12/19

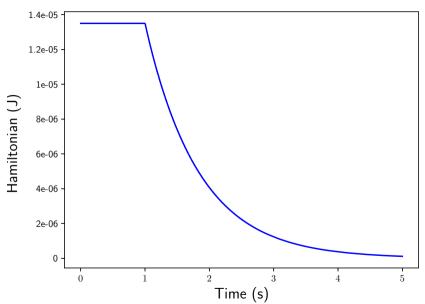
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Stabilization by boundary injection

Control parameter (
$$t_{\sf end} = 5[s]$$
)

$$K = \begin{cases} 0, & \forall t < 1 [s], \\ 100, & \forall t \ge 1 [s]. \end{cases}$$

Stabilization by boundary injection



The following has been presented:

³H. Blum and R. Rannacher. "On mixed finite element methods in plate bending analysis". In: *Computational Mechanics* 6.3 (1990), pp. 221–236. ISSN: 1432-0924. DOI: 10.1007/BF00350239.

⁴Katharina. Rafetseder and Walter. Zulehner. "A Decomposition Result for Kirchhoff Plate Bending Problems and a New Discretization Approach". In: *SIAM Journal on Numerical Analysis* 56.3 (2018), pp. 1961–1986. DOI: 10.1137/17M1118427.

The following has been presented:

• the Kirchhoff plate model as a port Hamiltonian system;

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Thanks for your attention Questions?

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