



THÈSE

En vue de l'obtention du

DOCTORAT DE L'UNIVERSITÉ DE TOULOUSE

Délivré par : *l'Institut Supérieur de l'Aéronautique et de l'Espace (ISAE)*

Présentée et soutenue le *30 Octobre 2020* par :

ANDREA BRUGNOLI

**A port-Hamiltonian formulation of flexible structures
Modelling and symplectic finite element discretization**

JURY

DANIEL ALAZARD	ISAE-Supaéro, Toulouse	Directeur
VALÉRIE P. BUDINGER	ISAE-Supaéro, Toulouse	Co-directeur
YANN LE GORREC	Institut FEMTO-ST	Rapporteur
ALESSANDRO MACCHELLI	Università di Bologna	Rapporteur
THOMAS HÉLIE	Directeur de Recherches CNRS	Examineur
?????	?????	Président

École doctorale et spécialité :

EDSYS : Automatique

Unité de Recherche :

CSDV - Commande des Systèmes et Dynamique du Vol - ONERA - ISAE

Directeur de Thèse :

Daniel ALAZARD et Valérie POMMIER-BUDINGER

Rapporteurs :

Yann LE GORREC et Alessandro MACCHELLI

Abstract

3 This thesis aims at extending the port-Hamiltonian (pH) approach to continuum mechanics
 4 in higher geometrical dimensions (particularly in 2D). The pH formalism has a strong mul-
 5 tiphysics character and represents a unified framework to model, analyze and control both
 6 finite- and infinite-dimensional systems. Despite the large literature on this topic, elasticity
 7 problems in higher geometrical dimensions have almost never been considered. This work
 8 establishes the connection between port-Hamiltonian distributed systems and elasticity prob-
 9 lems. The originality resides in three major contributions. First, the novel pH formulation
 10 of plate models and coupled thermoelastic phenomena is presented. The use of tensor cal-
 11 culus is mandatory for continuum mechanical models and the inclusion of tensor variables is
 12 necessary to obtain an intrinsic, i.e. coordinate free, and equivalent pH description. Second,
 13 a finite element based discretization technique, capable of preserving the structure of the
 14 infinite-dimensional problem at a discrete level, is developed and validated. The discretiza-
 15 tion of elasticity problems in port-Hamiltonian form requires the use of non-standard finite
 16 elements. Nevertheless, the numerical implementation is performed thanks to well-established
 17 open-source libraries, providing external users with an easy to use tool for simulating flexible
 18 systems in pH form. Third, flexible multibody systems are recast in pH form by making use of
 19 a floating frame description valid under small deformations assumptions. This reformulation
 20 include all kinds of linear elastic models and exploits the intrinsic modularity of pH systems.

22 Cette thèse vise à étendre l'approche port-hamiltonienne (pH) à la mécanique des milieux
 23 continus dans des dimensions géométriques plus élevées (en particulier on se focalise sur la
 24 dimension deux). Le formalisme pH, avec son fort caractère multiphysique, représente un
 25 cadre unifié pour modéliser, analyser et contrôler les systèmes de dimension finie et infinie.
 26 Malgré l'abondante littérature sur ce sujet, les problèmes d'élasticité en deux ou trois dimen-
 27 sions géométriques n'ont presque jamais été considérés. Dans ce travail de thèse la connexion
 28 entre problèmes d'élasticité et systèmes distribués port-Hamiltoniens est établie. L'originalité
 29 apportée réside dans trois contributions majeures. Tout d'abord, une nouvelle formu-
 30 lation pH des modèles de plaques et des phénomènes thermoélastiques couplés est présen-
 31 tée. L'utilisation du calcul tensoriel est obligatoire pour modéliser les milieux continus et
 32 l'introduction de variables tensorielles est nécessaire pour obtenir une description pH équiva-
 33 lente qui soit intrinsèque, c'est-à-dire indépendante des coordonnées choisies. Deuxièmement,
 34 une technique de discrétisation basée sur les éléments finis et capable de préserver la structure
 35 du problème de la dimension infinie au niveau discret est développée et validée. La discrétis-
 36 sation des problèmes d'élasticité écrits en forme port-Hamiltonienne nécessite l'utilisation
 37 d'éléments finis non standards. Néanmoins, l'implémentation numérique est réalisée grâce
 38 à des bibliothèques open source bien établies, fournissant aux utilisateurs externes un outil
 39 facile à utiliser pour simuler des systèmes flexibles sous forme pH. Troisièmement, une nou-
 40 velle formulation pH de la dynamique multicorps flexible est dérivée. Cette reformulation,
 41 valable sous de petites hypothèses de déformations, inclut toutes sortes de modèles élastiques
 42 linéaires et exploite la modularité intrinsèque des systèmes pH.

Acknowledgments

Remerciements

Ringraziamenti

Contents

48	Abstract	i
49	Résumé	iii
50	Acknowledgments	v
51	Remerciements	vii
52	Ringraziamenti	ix
53	List of Acronyms	xxi
54	I Introduction and state of the art	1
55	1 Introduction	3
56	1.1 Motivation and context	3
57	1.2 Overview of chapters	3
58	1.3 Contributions	3
59	2 Literature review	5
60	2.1 Port-Hamiltonian distributed systems	5
61	2.2 Structure-preserving discretization	5
62	2.3 Mixed finite element for elasticity	5
63	2.4 Multibody dynamics	5
64	3 Reminder on port-Hamiltonian systems	7
65	3.1 The Stokes-Dirac structure	8
66	3.1.1 Dirac Structures	8

67	3.1.2	Finite-dimensional port-Hamiltonian systems	9
68	3.1.3	Constant matrix differential operators	9
69	3.1.4	Constant Stokes-Dirac structures	11
70	3.2	Distributed port-Hamiltonian systems	12
71	3.2.1	Euler Bernoulli beam	14
72	3.2.2	Wave equation	15
73	3.2.3	2D shallow water equations	16
74	3.3	Conclusion	18
75	II	Port-Hamiltonian elasticity and thermoelasticity	19
76	4	Elasticity in port-Hamiltonian form	21
77	4.1	Continuum mechanics	21
78	4.1.1	Non linear formulation of elasticity	21
79	4.1.2	The linear elastodynamics problem	23
80	4.2	Port-Hamiltonian formulation of linear elasticity	25
81	4.2.1	Energy and co-energy variables	25
82	4.2.2	Final system and associated Stokes-Dirac structure	27
83	4.3	Conclusion	31
84	5	Port-Hamiltonian plate theory	33
85	5.1	First order plate theory	34
86	5.1.1	Mindlin-Reissner model	35
87	5.1.2	Kirchhoff-Love model	36
88	5.2	Port-Hamiltonian formulation of isotropic plates	38
89	5.2.1	Port-Hamiltonian Mindlin plate	39
90	5.2.2	Port-Hamiltonian Kirchhoff plate	43
91	5.3	Laminated anisotropic plates	48

92	5.3.1	Port-Hamiltonian laminated Mindlin plate	50
93	5.3.2	Port-Hamiltonian laminated Kirchhoff plate	51
94	5.4	Conclusion	52
95	6	Thermoelasticity in port-Hamiltonian form	55
96	6.1	Port-Hamiltonian linear coupled thermoelasticity	55
97	6.1.1	The heat equation as a pH descriptor system	56
98	6.1.2	Classical thermoelasticity	57
99	6.1.3	Thermoelasticity as two coupled pHs	59
100	6.2	Thermoelastic port-Hamiltonian bending	61
101	6.2.1	Thermoelastic Euler-Bernoulli beam	61
102	6.2.2	Thermoelastic Kirchhoff plate	63
103	6.3	Conclusion	65
104	III	Finite element structure preserving discretization	67
105	7	Partitioned finite element method	69
106	7.1	Discretization under uniform boundary condition	69
107	7.1.1	General procedure	71
108	7.1.2	Linear case	80
109	7.1.3	Linear flexible structures	82
110	7.2	Mixed boundary conditions	91
111	7.2.1	Solution using Lagrange multipliers	92
112	7.2.2	Virtual domain decomposition	94
113	7.3	Conclusion	98
114	8	Convergence numerical study	99
115	8.1	Plate problems using known mixed finite elements	99
116	8.2	Non-standard discretization of flexible structures	99

117	9 Numerical applications	101
118	9.1 Boundary stabilization	101
119	9.2 Thermoelastic wave propagation	101
120	9.3 Mixed boundary conditions	101
121	9.3.1 Trajectory tracking of a thin beam	101
122	9.3.2 Vibroacoustic under mixed boundary conditions	101
123	9.4 Modal analysis of plates	101
124	IV Port-Hamiltonian flexible multibody dynamics	103
125	10 Modular multibody systems in port-Hamiltonian form	105
126	10.1 Reminder of the rigid case	105
127	10.2 Flexible floating body	105
128	10.3 Modular construction of multibody systems	105
129	11 Validation	107
130	11.1 Beam systems	107
131	11.1.1 Modal analysis of a flexible mechanism	107
132	11.1.2 Non-linear crank slider	107
133	11.1.3 Hinged beam	107
134	11.2 Plate systems	107
135	11.2.1 Boundary interconnection with a rigid element	107
136	11.2.2 Actuated plate	107
137	Conclusions and future directions	111
138	A Mathematical tools	113
139	A.1 Differential operators	113
140	A.2 Integration by parts	114

141	A.3 Bilinear forms	115
142	B Finite elements gallery	117
143	C Implementation using FEniCS and Firedrake	119
144	Bibliography	121

List of Figures

146	4.1	A 2D continuum with Neumann and Dirichlet boundary conditions	29
147	5.1	Kinematic assumption for the Kirchhoff plate	37
148	5.2	Cauchy law for momenta and forces at the boundary.	40
149	5.3	Reference frames and notations.	40
150	5.4	Boundary conditions for the Mindlin plate.	41
151	5.5	Boundary conditions for the Kirchhoff plate.	46
152	5.6	Laminated plate with 4 layers.	48
153	6.1	Boundary conditions for the thermoelastic problem.	58
154	7.1	Partition of boundary into two connected sets.	92
155	7.2	Splitting of the domain into Ω_1, Ω_2	95

List of Tables

List of Acronyms

157

158	DAE	<i>Differential-Algebraic Equation</i>
159	dpHs	<i>distributed port-Hamiltonian systems</i>
160	FEM	<i>Finite Element Method</i>
161	IDA-PBC	<i>Interconnection and Damping Assignment Passivity Based Control</i>
162	PDE	<i>Partial Differential Equation</i>
163	PFEM	<i>Partitioned Finite Element Method</i>
164	pH	<i>port-Hamiltonian</i>
165	pHs	<i>port-Hamiltonian systems</i>
166	pHDAE	<i>port-Hamiltonian Descriptor System</i>

167

Part I

Introduction and state of the art

170

171

172

173

174
175
176
177
178
179
180
181

Introduction

I was born not knowing and have had only a little time to change that
here and there.

Richard Feynman
Letter to Armando Garcia J.

Contents

1.1	Motivation and context	3
1.2	Overview of chapters	3
1.3	Contributions	3

- 1.1 Motivation and context**
- 1.2 Overview of chapters**
- 1.3 Contributions**

Literature review

Books serve to show a man that those original thoughts of his aren't very new after all.

Abraham Lincoln

2.1 Port-Hamiltonian distributed systems

For 1D linear PH systems with a generalized skew-adjoint system operator, [LGZM05] gives conditions on the assignment of boundary inputs and outputs for the system operator to generate a contraction semigroup. The latter is instrumental to show well-posedness of a linear PH system, see [JZ12]. Essentially, at most half the number of boundary port variables can be imposed as control inputs for a well-posed PH system in 1D.

2.2 Structure-preserving discretization

2.3 Mixed finite element for elasticity

2.4 Multibody dynamics

Reminder on port-Hamiltonian systems

Contents

3.1	The Stokes-Dirac structure	8
3.1.1	Dirac Structures	8
3.1.2	Finite-dimensional port-Hamiltonian systems	9
3.1.3	Constant matrix differential operators	9
3.1.4	Constant Stokes-Dirac structures	11
3.2	Distributed port-Hamiltonian systems	12
3.2.1	Euler Bernoulli beam	14
3.2.2	Wave equation	15
3.2.3	2D shallow water equations	16
3.3	Conclusion	18



The main mathematical aspects behind the pH formalism are recalled in this chapter. First, the concept of Stokes-Dirac structure is presented. This notion was first introduced in the literature by making use of a differential geometry approach [vdSM02]. Despite being really insightful in terms of geometrical structure, this approach does not encompass the case of higher-order differential operators. An extension in this sense is still an open question. Since bending problems in elasticity introduce higher-order differential operators, the language of PDE will be privileged over the one of differential forms. To have the most suitable definition of Stokes-Dirac structure for flexible systems, the approach adopted in [MvdSM04] is here recovered.

Second, distributed port-Hamiltonian systems are introduced, in connection with the underlying Stokes-Dirac structure. PHs as boundary control systems have been analyzed deeply in one geometrical dimension [JZ12, LGZM05]. The complete characterization of pH in arbitrary dimension is still an open research field. Two notable exceptions [KZ15, Skr19] provide partial answers to this problem. The first demonstrate the well-posedness of the linear wave equation in arbitrary geometrical dimensions. The second generalizes this result to treat the case of generic first order linear pHs in arbitrary geometrical dimensions.

3.1 The Stokes-Dirac structure

In the section the concept of Stokes-Dirac structure for distributed, i.e. infinite-dimensional, pHs is introduced. First, the finite-dimensional case is considered. Then, to introduce the infinite-dimensional extension of Dirac structure, namely the Stokes-Dirac structure, the differential operators that come into play are characterized.

3.1.1 Dirac Structures

Consider a finite dimensional space F over the field \mathbb{R} and $E \equiv F'$ its dual, i.e. the space of linear operator $\mathbf{e} : F \rightarrow \mathbb{R}$. The elements of F are called flows, while the elements of E are called efforts. Those are port variables and their combination gives the power flowing inside the system. The space $B = F \times E$ is called the bond space of power variables. Therefore the power is defined as $\langle \mathbf{e}, \mathbf{f} \rangle = \mathbf{e}(\mathbf{f})$, where $\langle \mathbf{e}, \mathbf{f} \rangle$ is the dual product between \mathbf{f} and \mathbf{e} .

Definition 1 ([Cou90], Def. 1.1.1)

Given the finite-dimensional space F and its dual E with respect to the inner product $\langle \cdot, \cdot \rangle : F \times E \rightarrow \mathbb{R}$, consider the symmetric bilinear form:

$$\langle \langle (\mathbf{f}_1, \mathbf{e}_1), (\mathbf{f}_2, \mathbf{e}_2) \rangle \rangle := \langle \mathbf{e}_1, \mathbf{f}_2 \rangle + \langle \mathbf{e}_2, \mathbf{f}_1 \rangle, \quad \text{where} \quad \mathbf{f}_i, \mathbf{e}_i \in B, \quad i = 1, 2 \quad (3.1)$$

A Dirac structure on $B := F \times E$ is a subspace $D \subset B$, which is maximally isotropic under $\langle \langle \cdot, \cdot \rangle \rangle$. Equivalently, a Dirac structure on $B := F \times E$ is a subspace $D \subset B$ which equals its orthogonal complement with respect to $\langle \langle \cdot, \cdot \rangle \rangle : D = D^\perp$.

This definition can be extended to consider distributed forces and dissipation [Vil07].

Proposition 1

Consider the space of power variables $F \times E$ and let X denote an n -dimensional space, the space of energy variables. Suppose that $F := F_s \times F_e$ and that $E := E_s \times E_e$, with $\dim F_s = \dim E_s = n$ and $\dim F_e = \dim E_e = m$. Moreover, let $\mathbf{J}(\mathbf{x})$ denote a skew-symmetric matrix of dimension n and $\mathbf{B}(\mathbf{x})$ a matrix of dimension $n \times m$. Then, the set

$$D := \left\{ (\mathbf{f}_s, \mathbf{f}_e, \mathbf{e}_s, \mathbf{e}_e) \in F \times E \mid \mathbf{f}_s = -\mathbf{J}(\mathbf{x})\mathbf{e}_s - \mathbf{B}(\mathbf{x})\mathbf{f}_e, \mathbf{e}_e = \mathbf{B}(\mathbf{x})^\top \mathbf{e}_s \right\} \quad (3.2)$$

is a Dirac structure.

3.1.2 Finite-dimensional port-Hamiltonian systems

Consider the time-invariant dynamical system:

$$\begin{cases} \dot{\mathbf{x}} &= \mathbf{J}(\mathbf{x})\nabla H(\mathbf{x}) + \mathbf{B}(\mathbf{x})\mathbf{u}, \\ \mathbf{y} &= \mathbf{B}(\mathbf{x})^\top \nabla H(\mathbf{x}), \end{cases} \quad (3.3)$$

where $H(\mathbf{x}) : X \rightarrow \mathbb{R}$, the Hamiltonian, is a real-valued function bounded from below. Such a system is called port-Hamiltonian, as it arises from the Hamiltonian modelling of a physical system and it interacts with the environment through the input \mathbf{u} , included in the formulation. The connection with the concept of Dirac structure is achieved by considering the following port behavior:

$$\begin{aligned} \mathbf{f}_s &= -\dot{\mathbf{x}}, & \mathbf{e}_s &= \nabla H(\mathbf{x}), \\ \mathbf{f}_e &= \mathbf{u}, & \mathbf{e}_e &= \mathbf{y}. \end{aligned} \quad (3.4)$$

With this choice of the port variables, system (3.3) defines, by Proposition 1, a Dirac structure. Dissipation and distributed forces can be included and the corresponding system defines an extended Dirac structure, once the proper port variables have been introduced.

3.1.3 Constant matrix differential operators

Let Ω denote a compact subset of \mathbb{R}^d representing the spatial domain of the distributed parameter system. Then, let $U = C^\infty(\Omega, \mathbb{R}^{q_u})$ and $V = C^\infty(\Omega, \mathbb{R}^{q_v})$ denote the sets of smooth functions from Ω to \mathbb{R}^{q_u} and \mathbb{R}^{q_v} respectively.

Definition 2

A constant matrix differential operator of order n is a map $\mathcal{L} : U \rightarrow V$ such that, given $\mathbf{u} = (u_1, \dots, u_{q_u}) \in U$ and $\mathbf{v} = (v_1, \dots, v_{q_v}) \in V$:

$$\mathbf{v} = \mathcal{L}\mathbf{u} \iff \mathbf{v} := \sum_{|\alpha|=0}^n \mathbf{P}_\alpha \partial^\alpha \mathbf{u}, \quad (3.5)$$

where $\alpha := (\alpha_1, \dots, \alpha_d)$ is a multi-index of order $|\alpha| := \sum_{i=1}^d \alpha_i$, \mathbf{P}_α is a set of constant real $q_v \times q_u$ matrices and $\partial^\alpha := \partial_{x_1}^{\alpha_1} \dots \partial_{x_d}^{\alpha_d}$ is a differential operator of order $|\alpha|$ resulting from a combination of spatial derivatives.

The following definition, instrumental for the case of dpHs, is a simplified version of (6).

Definition 3

Consider the constant matrix differential operator (3.5). Its formal adjoint is the map \mathcal{L}^* from V to U such that:

$$\mathbf{u} = \mathcal{L}^*\mathbf{v} \iff \mathbf{u} := \sum_{|\alpha|=0}^n (-1)^{|\alpha|} \mathbf{P}_\alpha^\top \partial^\alpha \mathbf{v}. \quad (3.6)$$

Remark 1 (Differences between adjoint and formal adjoint)

The definition of formal adjoint is such that the integration by parts formula is respected

$$\int_{\Omega} \mathbf{a} \cdot (\mathcal{L}\mathbf{b}) \, d\Omega = \int_{\Omega} (\mathcal{L}^*\mathbf{a}) \cdot \mathbf{b} \, d\Omega,$$

where $\mathbf{a} \in C_0^\infty(\Omega, \mathbb{R}^{q_u})$, $\mathbf{b} \in C_0^\infty(\Omega, \mathbb{R}^{q_v})$ are smooth functions with compact support. This corresponds to the adjoint definition for a bounded operator between L^2 spaces of square integrable functions

$$\langle \mathbf{a}, \mathcal{L}\mathbf{b} \rangle_{L^2(\Omega, \mathbb{R}^{q_v})} = \langle \mathcal{L}^*\mathbf{a}, \mathbf{b} \rangle_{L^2(\Omega, \mathbb{R}^{q_u})}.$$

That means that, contrarily to the adjoint of an operator, the formal adjoint definition does not regard the actual domain of the operator nor the boundary conditions. For example, the differential operators div , grad are unbounded in the L^2 topology. Whenever unbounded operators are considered, it is important to define their domain. To avoid the need of specifying domains, the notion of formal adjoint is used. The formal adjoint respects the integration by parts formula and is defined only for sufficiently smooth functions with compact support. In this sense the formal adjoint of div is $-\text{grad}$, since for smooth functions with compact support, it holds

$$\langle \mathbf{y}, \text{grad } x \rangle_{L^2(\Omega, \mathbb{R}^3)} \underset{\text{I.B.P.}}{=} -\langle \text{div}(\mathbf{y}), x \rangle_{L^2(\Omega, \mathbb{R})},$$

for $\mathbf{y} \in C_0^\infty(\Omega, \mathbb{R}^n)$, $x \in C_0^\infty(\Omega)$ (I.B.P. stands for integration by parts). The definition of the domain of the operators, that requires the knowledge of the boundary conditions, has not been specified.

When $q_u = q_v = q \implies U \equiv V = W$, formal skew-adjoint operators can be defined:

Definition 4

Let $W = C^\infty(\Omega, \mathbb{R}^q)$ be the space of vector-valued smooth functions and $\mathcal{J} : W \rightarrow W$ a constant matrix differential operator. Then, \mathcal{J} is formally skew-adjoint (or skew-symmetric) if and only if $\mathcal{J} = -\mathcal{J}^*$. This corresponds to the algebraic condition on $q \times q$ square matrices

$$\mathbf{P}_\alpha = (-1)^{|\alpha|+1} \mathbf{P}_\alpha^\top, \quad \forall \alpha. \quad (3.7)$$

An important relation between a differential operator and its adjoint is expressed by the following theorem, valid for operators between spaces of different dimensions.

Theorem 1 ([RR04], Chapter 9, theorem 9.37)

Consider a matrix differential operator $\mathcal{L} : U \rightarrow V$ and let \mathcal{L}^* denote its formal adjoint. Then, for each function $\mathbf{u} \in U$ and $\mathbf{v} \in V$:

$$\int_{\Omega} (\mathbf{v}^\top \mathcal{L}\mathbf{u} - \mathbf{u}^\top \mathcal{L}^*\mathbf{v}) \, d\Omega = \int_{\partial\Omega} \tilde{\mathcal{A}}_{\mathcal{L}}(\mathbf{u}, \mathbf{v}) \, dS, \quad (3.8)$$

where $\tilde{\mathcal{A}}_{\mathcal{L}}$ is a differential operator induced on the boundary $\partial\Omega$ by \mathcal{L} , or equivalently:

$$\mathbf{v}^\top \mathcal{L} \mathbf{u} - \mathbf{u}^\top \mathcal{L}^* \mathbf{v} = \operatorname{div} \tilde{\mathcal{A}}_{\mathcal{L}}(\mathbf{u}, \mathbf{v}). \quad (3.9)$$

It is important to note that $\tilde{\mathcal{A}}_{\mathcal{L}}$ is a constant differential operator. The quantity $\tilde{\mathcal{A}}_{\mathcal{L}}(\mathbf{u}, \mathbf{v})$ is a constant linear combination of the functions \mathbf{u} and \mathbf{v} together with their spatial derivatives up to a certain order and depending on \mathcal{L} .

Corollary 1

Consider a skew-symmetric differential operator \mathcal{J} . For each function $\mathbf{u}, \mathbf{v} \in W = C^\infty(\Omega, \mathbb{R}^q)$ it holds:

$$\int_{\Omega} (\mathbf{v}^\top \mathcal{J} \mathbf{u} + \mathbf{u}^\top \mathcal{J} \mathbf{v}) \, d\Omega = \int_{\partial\Omega} \tilde{\mathcal{A}}_{\mathcal{J}}(\mathbf{u}, \mathbf{v}) \, dS, \quad (3.10)$$

where $\tilde{\mathcal{A}}_{\mathcal{J}}$ is a symmetric differential operator on $\partial\Omega$ depending on the differential operator \mathcal{J} .

3.1.4 Constant Stokes-Dirac structures

Following [MvdSM04], let F denote the space of flows, i.e. the space of smooth functions from the compact set $\Omega \subset \mathbb{R}^d$ to \mathbb{R}^q . For simplicity assume that the space of efforts is $E \equiv F$ (generally speaking these spaces are Hilbert spaces linked by duality, as in [Vil07]). Given $\mathbf{f} = (f_1, \dots, f_q) \in F$ and $\mathbf{e} = (e_1, \dots, e_q) \in E$. Let $\mathbf{z} = \mathcal{A}_{\partial}(\mathbf{e})$ denote the boundary terms, where \mathcal{A}_{∂} provides the restriction on $\partial\Omega$ of the effort variables \mathbf{e} and of their spatial derivatives of proper order. The associated boundary space is $Z := \{\mathbf{z} \mid \mathbf{z} = \mathcal{A}_{\partial}(\mathbf{e})\}$. Then, it holds

$$\int_{\partial\Omega} \tilde{\mathcal{A}}_{\mathcal{J}}(\mathbf{e}_1, \mathbf{e}_2) \, dS = \int_{\partial\Omega} \mathcal{A}_{\mathcal{J}}(\mathbf{z}_1, \mathbf{z}_2) \, dS, \quad \text{with} \quad \tilde{\mathcal{A}}_{\mathcal{J}}(\cdot, \cdot) = \mathcal{A}_{\mathcal{J}}(\mathcal{A}_{\partial}(\cdot), \mathcal{A}_{\partial}(\cdot)). \quad (3.11)$$

The following theorem characterizes Stokes-Dirac structures for pHs of arbitrary geometrical dimension and differential order.

Proposition 2 (Proposition 3.3 [MvdSM04])

Consider the space of power variables $B = F \times E \times Z$. The linear subspace $D \subset B$

$$D_{\mathcal{J}} = \{(\mathbf{f}, \mathbf{e}, \mathbf{z}) \in F \times E \times Z \mid \mathbf{f} = -\mathcal{J}\mathbf{e}, \mathbf{z} = \mathcal{A}_{\partial}(\mathbf{e})\}, \quad (3.12)$$

is a Stokes-Dirac structure on B with respect to the pairing

$$\langle\langle (\mathbf{f}^1, \mathbf{e}^1, \mathbf{z}^1), (\mathbf{f}^2, \mathbf{e}^2, \mathbf{z}^2) \rangle\rangle := \int_{\Omega} (\mathbf{e}^{1\top} \mathbf{f}^2 + \mathbf{e}^{2\top} \mathbf{f}^1) \, d\Omega + \int_{\partial\Omega} \mathcal{A}_{\mathcal{J}}(\mathbf{z}^1, \mathbf{z}^2) \, dS. \quad (3.13)$$

From this proposition, if $(\mathbf{f}, \mathbf{e}, \mathbf{z}) \in D_{\mathcal{J}}$, then $\langle\langle (\mathbf{f}, \mathbf{e}, \mathbf{z}), (\mathbf{f}, \mathbf{e}, \mathbf{z}) \rangle\rangle = 0$, that is

$$\int_{\Omega} \mathbf{e}^\top \mathbf{f} \, d\Omega + \frac{1}{2} \int_{\partial\Omega} \mathcal{A}_{\mathcal{J}}(\mathbf{z}, \mathbf{z}) \, dS = 0. \quad (3.14)$$

This relation expresses the power conservation property of the Stokes–Dirac structure. It states the relation between the variation of internal energy (the integral on the domain Ω) with the power flowing through the boundary (the integral over $\partial\Omega$). Thanks to the power conservation property dpHs always dispose of an associated Stokes–Dirac structure. This concept can be extended to consider dissipation or distributed forces. To this aim, it is necessary to include additional ports to account for the power exchange due to these effects (see Theorem 3.4 [MvdSM04]).

Remark 2

The constant Stokes–Dirac structure has been defined in case of smooth vector-valued functions for simplicity. The definition is indeed more general and encompasses the case of more complex functional spaces, in particular the L^2 space of square integrable functions. Linear elasticity for example is defined on a mixed function space of vector- and tensor-valued functions, cf. Sec §4.2.

3.2 Distributed port-Hamiltonian systems

A distributed lossless port-Hamiltonian system is defined by a set of variables that describes the unknowns, by a formally skew-adjoint differential operator, an energy functional and a set of boundary inputs and corresponding conjugated outputs. Such a system is described by the following set of equations

$$\begin{aligned}\frac{\partial \alpha}{\partial t} &= \mathcal{J}e, \\ e &:= \frac{\delta H}{\delta \alpha}, \\ \mathbf{u}_\partial &= \mathcal{B}_\partial e, \\ \mathbf{y}_\partial &= \mathcal{C}_\partial e,\end{aligned}\tag{3.15}$$

The unknowns α are called energy variables in the port-Hamiltonian framework, the formally skew-adjoint operator \mathcal{J} is named interconnection operator (see Def. 4 for a precise definition of formal skew adjointness). $\mathcal{B}_\partial, \mathcal{C}_\partial$ are boundary operators, that provide the boundary input \mathbf{u}_∂ and output \mathbf{y}_∂ [TW09, Chapter 4]. The variational derivative of the Hamiltonian defines the co-energy variables e .

Remark 3

It will become clear in this section that the effort variables of the Stokes–Dirac structure are indeed equivalent to the co-energy variables of the pH system. This justifies using the same notation for both.

Definition 5 (Variational derivative, Def. 4.1 in [Olv93])

Consider a functional $H(\alpha)$

$$H(\alpha) = \int_{\Omega} \mathcal{H}(\alpha) \, d\Omega.$$

Given a variation $\alpha = \bar{\alpha} + \eta \delta \alpha$ the variational derivative $\frac{\delta H}{\delta \alpha}$ is defined as

$$H(\bar{\alpha} + \eta \delta \alpha) = H(\bar{\alpha}) + \eta \int_{\Omega} \frac{\delta H}{\delta \alpha} \cdot \delta \alpha \, d\Omega + O(\eta^2).$$

Remark 4

If the integrand does not contain derivative of the argument α then the variational derivative is equal to the partial derivative of the Hamiltonian density \mathcal{H}

$$\frac{\delta H}{\delta \alpha} = \frac{\partial \mathcal{H}}{\partial \alpha}.$$

343 Lossless port-Hamiltonian systems possess a peculiar property: the energy rate is given
344 by the power due to the boundary ports $\mathbf{u}_{\partial}, \mathbf{y}_{\partial}$

$$\dot{H} = \int_{\Omega} \frac{\delta H}{\delta \alpha} \cdot \frac{\partial \alpha}{\partial t} \, d\Omega \stackrel{\text{Stokes theorem}}{=} \int_{\partial\Omega} \mathbf{u}_{\partial} \cdot \mathbf{y}_{\partial} \, dS \quad (3.16)$$

345 From the energy rate, the structural power balance is obtained

$$- \int_{\Omega} \frac{\delta H}{\delta \alpha} \cdot \frac{\partial \alpha}{\partial t} \, d\Omega + \int_{\partial\Omega} \mathbf{u}_{\partial} \cdot \mathbf{y}_{\partial} \, dS = 0 \quad (3.17)$$

From (3.14), it is clear by identification that $\mathcal{A}_{\mathcal{J}}(\mathbf{z}, \mathbf{z}) = 2 \mathbf{u}_{\partial} \cdot \mathbf{y}_{\partial}$. This means that the boundary space can be split into boundary input and output

$$Z := \{\mathbf{z} \mid \mathbf{z} = \mathcal{A}_{\partial}(\mathbf{e}) = (\mathbf{u}_{\partial}, \mathbf{y}_{\partial})\}$$

346 If the flow, effort and boundary variables are chosen to be

$$\mathbf{f} := -\partial_t \alpha, \quad \mathbf{e} := \delta \alpha H, \quad \mathbf{z} := (\mathbf{u}_{\partial}, \mathbf{y}_{\partial}), \quad (3.18)$$

347 then system (3.15) defines a Stokes-Dirac structure by Proposition 2. In this rather
348 informal treatment of dpHs, no rigorous characterization whatsoever has been introduced for
349 operators $\mathcal{J}, \mathcal{B}_{\partial}, \mathcal{C}_{\partial}$ in system (3.15). A formal characterization of these operators has been
350 given in [LGZM05] for pH of generic order only in one geometrical dimensional. In Chapter
351 7 the operator \mathcal{J} will be better characterize using an appropriate partition. By applying a
352 general integration by parts formula, the operators $\mathcal{B}_{\partial}, \mathcal{C}_{\partial}$ associated to \mathcal{J} can be defined as
353 well. The following examples clarifies this assertion for some known pHs.

3.2.1 Euler Bernoulli beam

The Euler-Bernoulli beam model consists of one PDE, describing the vertical displacement along the beam length:

$$\rho A(x) \frac{\partial^2 w}{\partial t^2}(x, t) + \frac{\partial^2}{\partial x^2} \left(EI(x) \frac{\partial^2 w}{\partial x^2} \right) = 0, \quad x \in \Omega = \{0, L\}, \quad (3.19)$$

where $w(x, t)$ is the transverse displacement of the beam. The coefficients $\rho(x)$, $A(x)$, $E(x)$ and $I(x)$ are the mass density, cross section, Young's modulus of elasticity and the moment of inertia of a cross section. The energy variables are then chosen as follows:

$$\alpha_w = \rho A(x) \frac{\partial w}{\partial t}(x, t), \quad \text{Linear Momentum}, \quad \alpha_\kappa = \frac{\partial^2 w}{\partial x^2}(x, t), \quad \text{Curvature}. \quad (3.20)$$

Those variables are collected in the vector $\alpha = (\alpha_w, \alpha_\kappa)^T$, so that the Hamiltonian can be written as a quadratic functional in the energy variables:

$$H = \frac{1}{2} \int_{\Omega} \left\{ \frac{1}{\rho A} \alpha_w^2 + EI \alpha_\kappa^2 \right\} d\Omega \quad (3.21)$$

The co-energy variables are found by computing the variational derivative of the Hamiltonian:

$$\begin{aligned} e_w &:= \frac{\delta H}{\delta \alpha_w} = \frac{\partial w}{\partial t}(x, t), & \text{Vertical velocity,} \\ e_\kappa &:= \frac{\delta H}{\delta \alpha_\kappa} = EI(x) \frac{\partial^2 w}{\partial x^2}(x, t), & \text{Flexural momentum.} \end{aligned} \quad (3.22)$$

The underlying interconnection structure is then found to be:

$$\frac{\partial}{\partial t} \begin{pmatrix} \alpha_w \\ \alpha_\kappa \end{pmatrix} = \begin{bmatrix} 0 & -\partial_{xx} \\ \partial_{xx} & 0 \end{bmatrix} \begin{pmatrix} e_w \\ e_\kappa \end{pmatrix}. \quad (3.23)$$

The power flow gives access to the boundary variables:

$$\begin{aligned} \dot{H} &= \int_{\Omega} \{e_w \partial_t \alpha_w + e_\kappa \partial_t \alpha_\kappa\} d\Omega, \\ &= \int_{\Omega} \{-e_w \partial_{xx} e_\kappa + e_\kappa \partial_{xx} e_w\} d\Omega, & \text{Integration by parts,} \\ &= \int_{\partial\Omega} \{-e_w \partial_x e_\kappa + e_\kappa \partial_x e_w\} ds = \langle -e_w, \partial_x e_\kappa \rangle_{\partial\Omega} + \langle e_\kappa, \partial_x e_w \rangle_{\partial\Omega} \end{aligned} \quad (3.24)$$

Since the system is of differential order two, two pairing appears, giving rise to four combination of uniform boundary causality

- First case $u_{\partial,1} = e_w$, $u_{\partial,2} = \partial_x e_w$, $y_{\partial,1} = -\partial_x e_\kappa$, $y_{\partial,2} = e_\kappa$.

This imposes the vertical $e_w := \partial_t w$ and angular velocity $\partial_x e_w := \partial_{xt} w$ as boundary

inputs. If the inputs are null a clamped boundary condition is obtained.

- Second case $u_{\partial,1} = e_w$, $u_{\partial,2} = e_\kappa$, $y_{\partial,1} = -\partial_x e_\kappa$, $y_{\partial,2} = \partial_x e_w$.
This imposes the vertical velocity and flexural momentum $e_\kappa := EI\partial_{xx}w$ as boundary inputs. Zero inputs lead to a simply supported condition is found.
- Third case $u_{\partial,1} = -\partial_x e_\kappa$, $u_{\partial,2} = e_\kappa$, $y_{\partial,1} = e_w$, $y_{\partial,2} = \partial_x e_w$.
This imposes the shear force $\partial_x e_\kappa := \partial_x(EI\partial_{xx}w)$ and flexural momentum as boundary inputs. Null inputs correspond to a free condition.
- Forth case $u_{\partial,1} = -\partial_x e_\kappa$, $u_{\partial,2} = \partial_x e_w$, $y_{\partial,1} = e_w$, $y_{\partial,2} = e_\kappa$.
This imposes the shear force and angular velocity as boundary inputs.

3.2.2 Wave equation

Given an open bounded connected set $\Omega \subset \mathbb{R}^2$ with Lipschitz continuous boundary $\partial\Omega$, the propagation of sound in air can be described by the following model [TRLGK18]

$$\begin{aligned}\chi_s \partial_t p(\mathbf{x}, t) &= -\operatorname{div} \mathbf{v}, \\ \mu_0 \partial_t \mathbf{v}(\mathbf{x}, t) &= -\operatorname{grad} p,\end{aligned}\tag{3.25}$$

where the scalar fields χ_s , μ_0 are the constant adiabatic compressibility factor and the steady state mass density respectively. The scalar field and vector field $p \in \mathbb{R}$, $\mathbf{v} \in \mathbb{R}^2$ represents the variation of pressure and velocity from the steady state. The Hamiltonian (total energy) reads

$$H = \frac{1}{2} \int_{\Omega} \left\{ \chi_s p^2 + \mu_0 \|\mathbf{v}\|^2 \right\} d\Omega.$$

To recast (3.25) in pH form the energy variables has to be introduced $\boldsymbol{\alpha} = [\alpha_p, \boldsymbol{\alpha}_v]^\top$

$$\alpha_p := \chi_s p, \quad \boldsymbol{\alpha}_v := \mu_0 \mathbf{v}.$$

The Hamiltonian is rewritten as

$$H = \frac{1}{2} \int_{\Omega} \left\{ \frac{1}{\chi_s} \alpha_p^2 + \frac{1}{\mu_0} \|\boldsymbol{\alpha}_v\|^2 \right\} d\Omega.$$

By definition, the co-energy are

$$e_p = \frac{\delta H}{\delta \alpha_p} = \frac{1}{\chi_s} \alpha_p = p, \quad e_v = \frac{\delta H}{\delta \boldsymbol{\alpha}_v} = \frac{1}{\mu_0} \boldsymbol{\alpha}_v = \mathbf{v}.$$

Equation (3.25) can be recast in port-Hamiltonian form

$$\frac{\partial}{\partial t} \begin{pmatrix} \alpha_p \\ \boldsymbol{\alpha}_v \end{pmatrix} = \begin{bmatrix} 0 & -\operatorname{div} \\ -\operatorname{grad} & \mathbf{0} \end{bmatrix} \begin{pmatrix} e_p \\ e_v \end{pmatrix}.$$

From the energy rate it is possible to identify the boundary variables.

$$\begin{aligned}
\dot{H} &= + \int_{\Omega} \{e_p \partial_t \alpha_p + \mathbf{e}_v \cdot \partial_t \boldsymbol{\alpha}_v\} \, d\Omega, \\
&= - \int_{\Omega} \{e_p \operatorname{div} \mathbf{e}_v + \mathbf{e}_v \cdot \operatorname{grad} e_p\} \, d\Omega, && \text{Chain rule,} \\
&= - \int_{\Omega} \operatorname{div}(e_p \mathbf{e}_v) \, d\Omega, && \text{Stokes theorem,} \\
&= - \int_{\partial\Omega} e_p \mathbf{e}_v \cdot \mathbf{n} \, dS = - \langle e_p, \mathbf{e}_v \cdot \mathbf{n} \rangle_{\partial\Omega}.
\end{aligned}$$

The boundary term $\langle e_p, \mathbf{e}_v \rangle_{\partial\Omega}$ pairs two power variables. One is taken as control input, the other plays the role of power-conjugated output. The assignment of these roles to the boundary power variables is referred to as causality of the boundary port [KML18],[Kot19, Chapter 2]. Under uniform causality assumption, either e_p or \mathbf{e}_v can assume the role of (distributed) boundary input, but not both. This leads to two possible selections:

- First case $u_{\partial} = e_p$, $y_{\partial} = \mathbf{e}_v \cdot \mathbf{n}$.

This imposes the variable $e_p := p$ as boundary input and corresponds to a classical Dirichlet condition.

- Second case $u_{\partial} = \mathbf{e}_v \cdot \mathbf{n}$, $y_{\partial} = e_p$.

This imposes the variable $\mathbf{e}_v \cdot \mathbf{n} := \mathbf{v} \cdot \mathbf{n}$ as boundary input and corresponds to a Neumann condition.

3.2.3 2D shallow water equations

This formulation may be found in [CR16, Section 6.2.]. This model describes a thin fluid layer of constant density in hydrostatic balance, like the propagation of a tsunami wave far from shore. Consider an open bounded connected set $\Omega \subset \mathbb{R}^2$ and a constant bed profile. The mass conservation implies

$$\frac{\partial h}{\partial t} + \operatorname{div}(h\mathbf{v}) = 0,$$

where $h(x, y, t) \in \mathbb{R}$ is a scalar field representing the fluid height, $\mathbf{v}(x, y, t) \in \mathbb{R}^2$ is the fluid velocity field. The conservation of linear momentum reads

$$\frac{\partial \rho \mathbf{v}}{\partial t} + \rho(\mathbf{v} \cdot \nabla) \mathbf{v} + \nabla(\rho g h) = 0,$$

where ρ is the mass density and g the gravitational acceleration constant. Using the identity

$$(\mathbf{v} \cdot \nabla) \mathbf{v} = \frac{1}{2} \nabla(\|\mathbf{v}\|^2) + (\nabla \times \mathbf{v}) \times \mathbf{v},$$

where $\nabla \times$ is the rotational of \mathbf{v} (also denoted $\operatorname{curl} \mathbf{v}$), the momentum is rearranged as follows

$$\frac{\partial \rho \mathbf{v}}{\partial t} = - \nabla \left(\frac{1}{2} \rho \|\mathbf{v}\|^2 + \rho g h \right) - \rho (\nabla \times \mathbf{v}) \times \mathbf{v}.$$

The last term on the right-hand side can be rewritten

$$\rho(\nabla \times \mathbf{v}) \times \mathbf{v} = \begin{bmatrix} 0 & -\rho\omega \\ \rho\omega & 0 \end{bmatrix} \mathbf{v},$$

with $\omega = \partial_x v_y - \partial_y v_x$ the local vorticity term. To derive a suitable pH formulation, the total energy, made up of kinetic and potential contribution, has to be invoked

$$H = \frac{1}{2} \int_{\Omega} \left\{ \rho h \|\mathbf{v}\|^2 + \rho g h^2 \right\} d\Omega.$$

394 As energy variable the fluid height and the linear momentum are chosen

$$\alpha_h = h, \quad \alpha_v = \rho \mathbf{v}. \quad (3.26)$$

395 The Hamiltonian is a non separable functional of the energy variables

$$H(\alpha_h, \alpha_v) = \frac{1}{2} \int_{\Omega} \left\{ \frac{1}{\rho} \alpha_h \|\alpha_v\|^2 + \rho g \alpha_h^2 \right\} d\Omega. \quad (3.27)$$

396 The co-energy variables are given by

$$e_h := \frac{\delta H}{\delta \alpha_h} = \frac{1}{2\rho} \|\alpha_v\|^2 + \rho g \alpha_h, \quad \mathbf{e}_v := \frac{\delta H}{\delta \alpha_v} = \frac{1}{\rho} \alpha_h \alpha_v. \quad (3.28)$$

397 The mass and momentum conservation are then rewritten as follows

$$\frac{\partial}{\partial t} \begin{pmatrix} \alpha_h \\ \alpha_v \end{pmatrix} = \begin{bmatrix} 0 & -\text{div} \\ -\text{grad} & \mathcal{G} \end{bmatrix} \begin{pmatrix} e_h \\ \mathbf{e}_v \end{pmatrix}, \quad (3.29)$$

The gyroscopic skew-symmetric term \mathcal{G} introduces a non-linearity as it depends on the energy variables

$$\mathcal{G}(\alpha_h, \alpha_v) = \frac{\omega}{\alpha_h} \begin{bmatrix} 0 & -1 \\ 1 & 0 \end{bmatrix}, \quad \omega = \partial_x \alpha_{v,y} - \partial_y \alpha_{v,x}.$$

398 Despite the non-standard formulation, the energy rate provides anyway the boundary vari-
399 ables

$$\begin{aligned} \dot{H} &= + \int_{\Omega} \{ e_h \partial_t \alpha_h + \mathbf{e}_v \cdot \partial_t \alpha_v \} d\Omega, \\ &= - \int_{\Omega} \{ e_h \text{div} \mathbf{e}_v + \mathbf{e}_v \cdot (\text{grad} e_h - \mathcal{G} \mathbf{e}_v) \} d\Omega, && \text{skew-symmetry of } \mathcal{G}, \\ &= - \int_{\Omega} \{ e_h \text{div} \mathbf{e}_v + \mathbf{e}_v \cdot \text{grad} e_h \} d\Omega, && \text{Chain rule,} \\ &= - \int_{\Omega} \text{div}(e_h \mathbf{e}_v) d\Omega, && \text{Stokes theorem,} \\ &= - \int_{\partial\Omega} e_h \mathbf{e}_v \cdot \mathbf{n} dS = - \langle e_h, \mathbf{e}_v \cdot \mathbf{n} \rangle_{\partial\Omega}. \end{aligned} \quad (3.30)$$

400 Again two possible cases of uniform boundary causality arise:

- First case $u_{\partial} = e_h$, $y_{\partial} = \mathbf{e}_v \cdot \mathbf{n}$.

This imposes the variable $e_h := h$ as boundary input and corresponds to a given water level for a fluid boundary.

- Second case $u_{\partial} = \mathbf{e}_v \cdot \mathbf{n}$, $y_{\partial} = e_p$.

This imposes the variable $\mathbf{e}_v \cdot \mathbf{n} := h\mathbf{v} \cdot \mathbf{n}$ as boundary input and corresponds to a given volumetric flow rate.

3.3 Conclusion

In this chapter, the main mathematical tools needed to understand infinite-dimensional pHs were recalled. A general characterization of the underlying operators behind a boundary control pH system is still an open topic. We have recalled some results available in the literature. Unfortunately, these do not provide a perfectly coherent treatment of pH systems of generic order on multi-dimensional domains. In Chapter 7, these operators are characterized, in connection to the discretization method developed.

Part II

Port-Hamiltonian elasticity and thermoelasticity

Elasticity in port-Hamiltonian form

I try not to break the rules but merely to test their elasticity.

Bill Veeck

Contents

4.1	Continuum mechanics	21
4.1.1	Non linear formulation of elasticity	21
4.1.2	The linear elastodynamics problem	23
4.2	Port-Hamiltonian formulation of linear elasticity	25
4.2.1	Energy and co-energy variables	25
4.2.2	Final system and associated Stokes-Dirac structure	27
4.3	Conclusion	31



Continuum mechanics is the mathematical description of how materials behave kinematically under external excitations. In this framework, the microscopic structure of a material body is neglected and a macroscopic viewpoint, that describes the body as a continuum, is adopted. This leads to a PDE based model. In this chapter, the general linear elastodynamics problem is recalled. A suitable port-Hamiltonian formulation is then derived.

4.1 Continuum mechanics

In this section, the main concepts behind a deformable continuum are briefly recalled following [Lee12]. For a detailed discussion on this topic, the reader may consult [Abe12, LPKL12].

4.1.1 Non linear formulation of elasticity

The bounded region of \mathbb{R}^d ($d = 2, 3$) occupied by a solid is called configuration. The reference configuration Ω is the domain that a bodies occupies at the initial state. To describe how the

body deforms in time the deformation map $\Phi : \Omega \times [0, T_f] \rightarrow \Omega' \subset \mathbb{R}^d$ is introduced. This map is differentiable and orientation preserving, and the image of Ω under $\Phi(\cdot, t) \forall t \in [0, T_f]$ is called the deformed configuration Ω_t . Given a specific point in the reference frame its image is denoted by $\mathbf{y} = \Phi(\mathbf{x}, t)$. The gradient of the deformation map is called the deformation gradient $\mathbf{F} := \nabla_{\mathbf{x}} \Phi = \frac{\partial \mathbf{y}}{\partial \mathbf{x}}$. A rigid deformation maps a point $\mathbf{x} \in \Omega \rightarrow \mathbf{A}(t)\mathbf{x} + \mathbf{b}(t)$, where $\mathbf{A}(t)$ is an orthogonal matrix and $\mathbf{b}(t) \in \mathbb{R}^d$ a vector. A differentiable deformation map Φ is a rigid deformation iff $\mathbf{F}^\top \mathbf{F} - \mathbf{I} = 0$, where \mathbf{I} is the identity in $\mathbb{R}^{d \times d}$ (for the proof see [Cia88], page 44). For this reason, a suitable measure of the deformation is the Green-St.Venant strain tensor $\frac{1}{2}(\mathbf{F}^\top \mathbf{F} - \mathbf{I})$.

A quantity of interest is the displacement $\mathbf{u} : \Omega \times [0, T_f] \rightarrow \mathbb{R}^d$ with respect to the reference configuration. It is defined as $\mathbf{u}(\mathbf{x}, t) = \Phi(\mathbf{x}, t) - \mathbf{x}$. The gradient of the displacement verifies $\nabla_{\mathbf{x}} \mathbf{u} = \mathbf{F} - \mathbf{I}$. The strain tensor can now be written in terms of the displacement

$$\begin{aligned} \frac{1}{2}(\mathbf{F}^\top \mathbf{F} - \mathbf{I}) &= \frac{1}{2} \left[(\nabla_{\mathbf{x}} \mathbf{u} + \mathbf{I})^\top (\nabla_{\mathbf{x}} \mathbf{u} + \mathbf{I}) - \mathbf{I} \right] \\ &= \frac{1}{2} \left[\nabla_{\mathbf{x}} \mathbf{u} + (\nabla_{\mathbf{x}} \mathbf{u})^\top + (\nabla_{\mathbf{x}} \mathbf{u})^\top (\nabla_{\mathbf{x}} \mathbf{u}) \right], \end{aligned}$$

or in components

$$\frac{1}{2}(F_{ik}^\top F_{kj} - I_{ij}) = \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} + \frac{\partial u_i}{\partial x_j} \frac{\partial u_j}{\partial x_i} \right).$$

To state the balance laws the actual deformed configuration is considered. The linear and angular momenta in a subdomain $\omega_t \subset \Omega_t$ are computed as

$$\int_{\omega_t} \rho \mathbf{v} \, d\omega_t, \quad \text{and} \quad \int_{\omega_t} \rho \mathbf{y} \times \mathbf{v} \, d\omega_t,$$

where ρ is the mass density and the velocity $\mathbf{v} = \frac{D\mathbf{u}}{Dt}(\mathbf{y}, t) = \frac{\partial \mathbf{u}}{\partial t}(\mathbf{x}, t)$ is the material time derivative of the displacement (see [Abe12, Chapter 1]). Let $\omega_{t,1}, \omega_{t,2}$ be two subregions in a deformed continuum Ω_t with contacting surface S_{12} . There is a force acting on this surface for a continuum that is called stress vector or traction. If \mathbf{n} is the outward normal at \mathbf{y} on S_{12} with respect to $\omega_{t,1}$, then the surface force that $\omega_{t,1}$ exerts on $\omega_{t,2}$ is denoted by $\mathbf{t}(\mathbf{y}, \mathbf{n}) \in \mathbb{R}^d$. By the Newton third law, the surface force that $\omega_{t,2}$ applies on $\omega_{t,1}$ is given by $\mathbf{t}(\mathbf{y}, -\mathbf{n}) = -\mathbf{t}(\mathbf{y}, \mathbf{n})$. It is assumed that the linear and angular momentum balance hold for any subregion $\omega_t \in \Omega_t$

$$\begin{aligned} \frac{d}{dt} \int_{\omega_t} \rho \mathbf{v} \, d\omega_t &= \int_{\partial \omega_t} \mathbf{t}(\mathbf{y}, \mathbf{n}) \, dS + \int_{\omega_t} \mathbf{f} \, d\omega_t, \\ \frac{d}{dt} \int_{\omega_t} \rho \mathbf{y} \times \mathbf{v} \, d\omega_t &= \int_{\partial \omega_t} \mathbf{y} \times \mathbf{t}(\mathbf{y}, \mathbf{n}) \, dS + \int_{\omega_t} \mathbf{y} \times \mathbf{f} \, d\omega_t, \end{aligned}$$

443 where $\partial \omega_t$ stands for the boundary surface of the subdomain ω_t , \mathbf{n} is the outward normal to
 444 the surface $\partial \omega_t$ and \mathbf{f} represents an exterior body force. The following theorem characterizes
 445 the stress vector (see [Cia88, Chapter 2]):

Theorem 2 (Cauchy's theorem)

If the linear and angular momenta balance hold, then there exists a matrix-valued function Σ from Ω_t to \mathbb{S} such that $\mathbf{t}(\mathbf{y}, \mathbf{n}) = \Sigma(\mathbf{y})\mathbf{n}$, $\forall \mathbf{y} \in \Omega_t$ where the right-hand side is the matrix-vector multiplication.

The set $\mathbb{S} = \mathbb{R}_{\text{sym}}^{d \times d}$ denotes the field of symmetric matrices in $\mathbb{R}^{d \times d}$. The symmetry of the stress tensor Σ is due to the balance of angular momentum. The divergence theorem can then be applied

$$\int_{\partial\omega_t} \Sigma \mathbf{n} \, dS = \int_{\omega_t} \nabla_{\mathbf{y}} \cdot \Sigma \, d\omega_t,$$

where $\nabla_{\mathbf{y}} \cdot$ is the tensor divergence with respect to the deformed configuration, $\nabla_{\mathbf{y}} \cdot \Sigma = \sum_{i=1}^d \frac{\partial \Sigma_{ij}}{\partial y_i}$. Because the considered subregion ω_t is arbitrary, using the linear balance momentum and the conservation of mass, the following PDE is found

$$\rho \frac{D\mathbf{v}}{Dt} - \nabla_{\mathbf{y}} \cdot \Sigma = \mathbf{f}, \quad \mathbf{y} \in \Omega_t.$$

This equation is written with respect to the deformed configuration Ω_t . For a detailed derivation of this equation the reader may consult [Abe12, Chapter 4]. To obtain a closed formulation, the constitutive law, namely the link between Σ and the strain tensor $\frac{1}{2}(\mathbf{F}^\top \mathbf{F} - \mathbf{I})$, has to be introduced. In the next section such relation will be discussed for the case of linear elasticity.

4.1.2 The linear elastodynamics problem

Whenever deformations are small, $\|\nabla_{\mathbf{x}} \mathbf{u}\| \ll 1$, then the reference and deformed configurations are almost indistinguishable $\mathbf{y} = \mathbf{x} + \mathbf{u} = \mathbf{x} + O(\nabla_{\mathbf{x}} \mathbf{u}) \approx \mathbf{x}$. This allows writing the linear momentum balance in the reference configuration

$$\rho \frac{\partial \mathbf{v}}{\partial t}(\mathbf{x}, t) - \text{Div } \Sigma(\mathbf{x}, t) = \mathbf{f}, \quad \mathbf{x} \in \Omega.$$

The material derivative simplifies to a partial one. The operator Div is the divergence of a tensor field with respect to the reference configuration (see Appendix A for a description of the differential operators)

$$\text{Div } \Sigma(\mathbf{x}, t) = \nabla_{\mathbf{x}} \cdot \Sigma(\mathbf{x}, t) = \left(\sum_{i=1}^d \frac{\partial \Sigma_{ij}}{\partial x_i} \right)_{1 \leq j \leq d}.$$

Furthermore, the non-linear terms in the Green-St. Venant strain tensor can be dropped

$$\frac{1}{2}(\mathbf{F}^\top \mathbf{F} - \mathbf{I}) = \frac{1}{2} \left[\nabla_{\mathbf{x}} \mathbf{u} + (\nabla_{\mathbf{x}} \mathbf{u})^\top + (\nabla_{\mathbf{x}} \mathbf{u})^\top (\nabla_{\mathbf{x}} \mathbf{u}) \right] \approx \frac{1}{2} \left[\nabla_{\mathbf{x}} \mathbf{u} + (\nabla_{\mathbf{x}} \mathbf{u})^\top \right].$$

456 The linearized strain tensor (also called infinitesimal strain tensor) is the symmetric gradient
457 of the displacement

$$\boldsymbol{\varepsilon} := \text{Grad } \mathbf{u}, \quad \text{where} \quad \text{Grad } \mathbf{u} = \frac{1}{2} \left[\nabla_x \mathbf{u} + (\nabla_x \mathbf{u})^\top \right]. \quad (4.1)$$

To obtain a closed system of equations, it is now necessary to characterize the relation between stress and strain. This relation is normally called *constitutive law*. In the following, the particular case of elastic materials is considered. These are able to resist distorting excitations and return to its original size and shape when these excitations are removed. For this class of materials, the stress tensor is solely determined by the deformed configuration at a given time (Hooke's law)

$$\boldsymbol{\Sigma}(\mathbf{x}) = \boldsymbol{\mathcal{D}}(\mathbf{x}) \boldsymbol{\varepsilon}(\mathbf{u}(\mathbf{x})).$$

The *stiffness tensor* or *elasticity tensor* $\boldsymbol{\mathcal{D}} : \mathbb{S} \rightarrow \mathbb{S}$ is a rank 4 tensor that is symmetric positive definite and uniformly bounded above and below. Because of symmetry, its components satisfy

$$\mathcal{D}_{ijkl} = \mathcal{D}_{jikl} = \mathcal{D}_{klij}.$$

458 From the uniform boundedness of $\boldsymbol{\mathcal{D}}$, the map $\boldsymbol{\mathcal{D}} : L^2(\Omega; \mathbb{S}) \rightarrow L^2(\Omega; \mathbb{S})$ is a symmetric positive
459 definite bounded linear operator ($L^2(\Omega; \mathbb{S})$ is the space of square integrable symmetric tensor-
460 valued functions). The compliance tensor $\boldsymbol{\mathcal{C}}$ is defined by $\boldsymbol{\mathcal{C}} = \boldsymbol{\mathcal{D}}^{-1}$. Thus $\boldsymbol{\mathcal{C}} : \mathbb{S} \rightarrow \mathbb{S}$ is as
461 well symmetric positive definite and uniformly bounded above and below. An isotropic elastic
462 medium has the same kinematic properties in any direction and at each point. If an elastic
463 medium is isotropic, then the stiffness and compliance tensors assume the form

$$\boldsymbol{\mathcal{D}}(\cdot) = 2\mu(\cdot) \mathbf{I} + \lambda \text{Tr}(\cdot) \mathbf{I}, \quad \boldsymbol{\mathcal{C}}(\cdot) = \frac{1}{2\mu} \left[(\cdot) - \frac{\lambda}{2\mu + d\lambda} \text{Tr}(\cdot) \mathbf{I} \right], \quad d = \{2, 3\}, \quad (4.2)$$

464 where Tr is the trace operator and the positive scalar functions μ, λ , defined on Ω , are called
465 the Lamé coefficients. In engineering applications it is easier to compute experimentally two
466 other parameters: the Young modulus E and Poisson's ratio ν . Those are expressed in terms
467 of the Lamé coefficients as

$$\nu = \frac{\lambda}{2(\lambda + \mu)}, \quad E = \frac{\mu(3\lambda + 2\mu)}{\lambda + \mu}, \quad (4.3)$$

468 and conversely

$$\lambda = \frac{E\nu}{(1 + \nu)(1 - 2\nu)}, \quad \mu = \frac{E}{2(1 + \nu)}. \quad (4.4)$$

The stiffness and compliant tensor are expressed as

$$\boldsymbol{\mathcal{D}}(\cdot) = \frac{E}{1 + \nu} \left[(\cdot) + \frac{\nu}{1 - 2\nu} \text{Tr}(\cdot) \mathbf{I} \right], \quad (4.5)$$

$$\boldsymbol{\mathcal{C}}(\cdot) = \frac{1 + \nu}{E} \left[(\cdot) - \frac{\nu}{1 + \nu(d - 2)} \text{Tr}(\cdot) \mathbf{I} \right]. \quad (4.6)$$

The linear elastodynamics problem is formulated through a vector-valued PDE

$$\rho \frac{\partial^2 \mathbf{u}}{\partial t^2} - \text{Div}(\mathcal{D} \text{Grad } \mathbf{u}) = \mathbf{f}. \quad (4.7)$$

The classical elastodynamics problem is expressed considering the displacement \mathbf{u} as the unknown. This PDE goes together with appropriate boundary conditions that will be specified in 4.2.

4.2 Port-Hamiltonian formulation of linear elasticity

In this section a port-Hamiltonian formulation for elasticity is deduced from the classical elastodynamics problem. It must be highlighted that already in the seventies a purely hyperbolic formulation for elasticity was detailed [HM78]. The missing point is the clear connection with the theory of Hamiltonian PDEs. An Hamiltonian formulation can be found in [Gri15, Chapter 16], but without any connection to the concept of Stokes-Dirac structure induced by the underlying geometry.

4.2.1 Energy and co-energy variables

Consider an open connected set $\Omega \subset \mathbb{R}^d$, $d = (2, 3)$. The displacement within a deformable continuum is given by Eq. (4.7).

$$\rho \frac{\partial^2 \mathbf{u}}{\partial t^2} - \text{Div}(\mathcal{D} \text{Grad } \mathbf{u}) = 0, \quad \mathbf{x} \in \Omega. \quad (4.8)$$

The contribution of the body force \mathbf{f} has been removed for ease of presentation. To derive a pH formulation, the total energy, that includes the kinetic and deformation energy, is needed

$$H = \frac{1}{2} \int_{\Omega} \left\{ \rho \|\partial_t \mathbf{u}\|^2 + \mathbf{\Sigma} : \mathbf{\varepsilon} \right\} d\Omega. \quad (4.9)$$

The notation $\mathbf{A} : \mathbf{B} = \text{Tr}(\mathbf{A}^\top \mathbf{B}) = \sum_{i,j} A_{ij} B_{ij}$ denotes the tensor contraction. Recall that $\mathbf{\varepsilon} = \text{Grad } \mathbf{u}$ and $\mathbf{\Sigma} = \mathcal{D} \mathbf{\varepsilon}$. The energy variables are then the linear momentum and the deformation field

$$\boldsymbol{\alpha}_v = \rho \mathbf{v}, \quad \mathbf{A}_\varepsilon = \mathbf{\varepsilon},$$

where $\mathbf{v} := \partial_t \mathbf{u}$. The Hamiltonian can be rewritten as a quadratic functional in the energy variables

$$H = \frac{1}{2} \int_{\Omega} \left\{ \frac{1}{\rho} \boldsymbol{\alpha}_v^2 + (\mathcal{D} \mathbf{A}_\varepsilon) : \mathbf{A}_\varepsilon \right\} d\Omega. \quad (4.10)$$

The co-energy variables are given by

$$\mathbf{e}_v := \frac{\delta H}{\delta \boldsymbol{\alpha}_v} = \mathbf{v}, \quad \mathbf{E}_\varepsilon := \frac{\delta H}{\delta \mathbf{A}_\varepsilon} = \mathbf{\Sigma}. \quad (4.11)$$

The tensor-valued co-energy \mathbf{E}_ε is obtained by taking the variational derivative with respect to a tensor.

Proposition 3

The variational derivative of the Hamiltonian with respect to the strain tensor is the stress tensor $\delta_{\mathbf{A}_\varepsilon} H = \boldsymbol{\Sigma}$.

Proof. Let $\mathbb{S} : \mathbb{R}_{\text{sym}}^{d \times d}$ be the space of symmetric tensor and $L^2(\Omega, \mathbb{S})$ the space of the square integrable symmetric tensors endowed with the tensor contraction as inner product

$$\langle \mathbf{A}, \mathbf{B} \rangle_{L^2(\Omega, \mathbb{S})} = \int_{\Omega} \mathbf{A} : \mathbf{B} \, d\Omega. \quad (4.12)$$

The contribution due to the deformation part in Hamiltonian is given by:

$$H_{\text{def}}(\mathbf{A}_\varepsilon) = \frac{1}{2} \int_{\Omega} (\mathcal{D} \mathbf{A}_\varepsilon) : \mathbf{A}_\varepsilon \, d\Omega.$$

A variation $\Delta \mathbf{A}_\varepsilon$ of the strain tensor with respect to a given value $\bar{\mathbf{A}}_\varepsilon$ leads to:

$$\begin{aligned} H_{\text{def}}(\bar{\mathbf{A}}_\varepsilon + \eta \Delta \mathbf{A}_\varepsilon) &= + \frac{1}{2} \int_{\Omega} (\mathcal{D} \bar{\mathbf{A}}_\varepsilon) : \bar{\mathbf{A}}_\varepsilon \, d\Omega \\ &+ \eta \frac{1}{2} \int_{\Omega} \left\{ (\mathcal{D} \bar{\mathbf{A}}_\varepsilon) : \Delta \mathbf{A}_\varepsilon + (\mathcal{D} \Delta \mathbf{A}_\varepsilon) : \bar{\mathbf{A}}_\varepsilon \right\} \, d\Omega + O(\eta^2). \end{aligned}$$

The term $(\mathcal{D} \Delta \mathbf{A}_\varepsilon) : \bar{\mathbf{A}}_\varepsilon$ can be further rearranged using the symmetry of \mathcal{D} and the commutativity of the tensor contraction

$$(\mathcal{D} \Delta \mathbf{A}_\varepsilon) : \bar{\mathbf{A}}_\varepsilon = (\mathcal{D} \bar{\mathbf{A}}_\varepsilon) : \Delta \mathbf{A}_\varepsilon,$$

so that

$$H_{\text{def}}(\bar{\mathbf{A}}_\varepsilon + \eta \Delta \mathbf{A}_\varepsilon) = \frac{1}{2} \int_{\Omega} (\mathcal{D} \bar{\mathbf{A}}_\varepsilon) : \bar{\mathbf{A}}_\varepsilon \, d\Omega + \eta \int_{\Omega} (\mathcal{D} \bar{\mathbf{A}}_\varepsilon) : \Delta \mathbf{A}_\varepsilon \, d\Omega + O(\eta^2).$$

By definition of variational derivative it can be written:

$$H_{\text{def}}(\bar{\mathbf{A}}_\varepsilon + \eta \Delta \mathbf{A}_\varepsilon) = H_{\text{def}}(\bar{\mathbf{A}}_\varepsilon) + \eta \left\langle \frac{\delta H}{\delta \mathbf{A}_\varepsilon}, \Delta \mathbf{A}_\varepsilon \right\rangle_{L^2(\Omega, \mathbb{S})} + O(\eta^2),$$

Then, by identification

$$\frac{\delta H_{\text{def}}}{\delta \mathbf{A}_\varepsilon} = \mathcal{D} \bar{\mathbf{A}}_\varepsilon = \boldsymbol{\Sigma}.$$

Since the Hamiltonian is separable then $\delta_{\mathbf{A}_\varepsilon} H_{\text{def}} = \delta_{\mathbf{A}_\varepsilon} H$, leading to the final result. \square

4.2.2 Final system and associated Stokes-Dirac structure

It is now possible to state the final pH form

$$\frac{\partial}{\partial t} \begin{pmatrix} \boldsymbol{\alpha}_v \\ \mathbf{A}_\varepsilon \end{pmatrix} = \begin{bmatrix} \mathbf{0} & \text{Div} \\ \text{Grad} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_v \\ \mathbf{E}_\varepsilon \end{pmatrix}. \quad (4.13)$$

The first equation of the system is the conservation of linear momentum. The second represents a compatibility condition

$$\begin{aligned} \partial_t \mathbf{A}_\varepsilon &= \text{Grad}(\mathbf{e}_v), \\ \partial_t \boldsymbol{\varepsilon} &= \text{Grad}(\mathbf{v}), \\ \partial_t \text{Grad } \mathbf{u} &= \text{Grad}(\partial_t \mathbf{u}). \end{aligned} \quad (4.14)$$

Assuming that $\mathbf{u} \in C^2$, higher order derivatives commute (Schwarz theorem). Hence, the equation is verified. The following theorem ensures the differential operator is formally skew-adjoint (one can also find this result in the recent article [PZ20, Lemma 3.3], available as arXiv preprint).

Theorem 3

The formal adjoint of the tensor divergence Div is $-\text{Grad}$, the opposite of the symmetric gradient.

Proof. We denote by $\mathbb{V} = \mathbb{R}^d$ the space of vector field in \mathbb{R}^d and by $\mathbb{S} = \mathbb{R}^{d \times d}$ the space of symmetric tensor field in $\mathbb{R}^{d \times d}$. Let us consider the Hilbert space of the square integrable symmetric tensors $L^2(\Omega, \mathbb{S})$ with scalar product is defined in (4.12). Moreover consider the Hilbert space of the square integrable vector function $L^2(\Omega, \mathbb{V})$, endowed with the usual scalar product:

$$\langle \mathbf{a}, \mathbf{b} \rangle_{L^2(\Omega, \mathbb{V})} = \int_{\Omega} \mathbf{a} \cdot \mathbf{b} \, d\Omega = \int_{\Omega} \mathbf{a}^\top \mathbf{b} \, d\Omega, \quad \forall \mathbf{a}, \mathbf{b} \in L^2(\Omega, \mathbb{V}).$$

Let us consider the tensor divergence operator defined as:

$$\begin{aligned} \text{Div} : L^2(\Omega, \mathbb{S}) &\rightarrow L^2(\Omega, \mathbb{V}), \\ \boldsymbol{\Psi} &\rightarrow \text{Div } \boldsymbol{\Psi} = \boldsymbol{\psi}, \end{aligned} \quad \text{with } \psi_j = \text{div}(\Psi_{ij}) = \sum_{i=1}^d \frac{\partial \Psi_{ij}}{\partial x_i}.$$

We try to identify Div^*

$$\begin{aligned} \text{Div}^* : L^2(\Omega, \mathbb{V}) &\rightarrow L^2(\Omega, \mathbb{S}), \\ \boldsymbol{\phi} &\rightarrow \text{Div}^* \boldsymbol{\phi} = \boldsymbol{\Phi}, \end{aligned}$$

such that

$$\begin{aligned} \langle \text{Div } \boldsymbol{\Psi}, \boldsymbol{\phi} \rangle_{L^2(\Omega, \mathbb{V})} &= \langle \boldsymbol{\Psi}, \text{Div}^* \boldsymbol{\phi} \rangle_{L^2(\Omega, \mathbb{S})}, & \forall \boldsymbol{\Psi} \in \text{Dom}(\text{Div}) \subset L^2(\Omega, \mathbb{S}) \\ & & \forall \boldsymbol{\phi} \in \text{Dom}(\text{Div}^*) \subset L^2(\Omega, \mathbb{V}) \end{aligned}$$

Now let us take $\boldsymbol{\Psi} \in C_0^1(\Omega, \mathbb{S}) \subset \text{Domain}(\text{Div})$ the space of differentiable symmetric tensors

with compact support in Ω . Additionally ϕ will belong to $C_0^1(\Omega, \mathbb{V}) \subset \text{Dom}(\text{Div}^*)$, the space of differentiable vector functions with compact support in Ω . Then

$$\begin{aligned}
 \langle \text{Div } \Psi, \phi \rangle_{L^2(\Omega, \mathbb{V})} &= \int_{\Omega} \psi \cdot \phi \, d\Omega, \\
 &= \int_{\Omega} \sum_{i=1}^d \sum_{j=1}^d \frac{\partial \Psi_{ij}}{\partial x_i} \phi_j \, d\Omega, \\
 &= - \int_{\Omega} \sum_{i=1}^d \sum_{j=1}^d \Psi_{ij} \frac{\partial \phi_j}{\partial x_i} \, d\Omega, \quad \text{since the functions vanish at the boundary,} \\
 &= - \int_{\Omega} \sum_{i=1}^d \sum_{j=1}^d \Psi_{ij} F_{ij} \, d\Omega, \quad \text{where } F_{ij} = \frac{\partial \phi_j}{\partial x_i}, \\
 &= - \langle \Psi, \mathbf{F} \rangle_{L^2(\Omega, \mathbb{S})}, \quad \mathbf{F} = \text{grad } \phi.
 \end{aligned}$$

508 But in this latter case, it could not be stated that $\mathbf{F} \in L^2(\Omega, \mathbb{S})$. Now, since $\Psi \in L^2(\Omega, \mathbb{S})$,
 509 $\Psi_{ji} = \Psi_{ij}$, thus the last equality can be further decomposed as

$$\sum_{i,j} \Psi_{ij} \frac{\partial \phi_j}{\partial x_i} = \sum_{i,j} \Psi_{ij} \frac{1}{2} \left(\frac{\partial \phi_i}{\partial x_j} + \frac{\partial \phi_j}{\partial x_i} \right) = \sum_{i,j} \Psi_{ij} \Phi_{ij}, \quad \text{with } \Phi_{ij} := \frac{1}{2} \left(\frac{\partial \phi_i}{\partial x_j} + \frac{\partial \phi_j}{\partial x_i} \right).$$

Thus $\Phi = \text{Grad } \phi \in L^2(\Omega, \mathbb{S})$ and it can be stated that:

$$\begin{aligned}
 \langle \text{Div } \Psi, \phi \rangle_{L^2(\Omega, \mathbb{V})} &= - \int_{\Omega} \sum_{i,j} \Psi_{ij} \frac{1}{2} \left(\frac{\partial \phi_i}{\partial x_j} + \frac{\partial \phi_j}{\partial x_i} \right) \, d\Omega \\
 &= - \int_{\Omega} \sum_{i,j} \Psi_{ij} \Phi_{ij} \, d\Omega = \langle \Psi, -\text{Grad } \phi \rangle_{L^2(\Omega, \mathbb{S})}.
 \end{aligned}$$

510 It can be concluded that the formal adjoint of Div is $\text{Div}^* = -\text{Grad}$. □

511 The boundary values are then found by evaluating the energy rate

$$\begin{aligned}
 \dot{H} &= \int_{\Omega} \{ \mathbf{e}_v \cdot \partial_t \boldsymbol{\alpha}_v + \mathbf{E}_{\varepsilon} : \partial_t \mathbf{A}_{\varepsilon} \} \, d\Omega, \\
 &= \int_{\Omega} \{ \mathbf{e}_v \cdot \text{Div } \mathbf{E}_{\varepsilon} + \mathbf{E}_{\varepsilon} : \text{Grad } \mathbf{e}_v \} \, d\Omega, \\
 &= \int_{\Omega} \text{div}(\mathbf{E}_{\varepsilon} \mathbf{e}_v) \, d\Omega, \quad \text{Stokes theorem (see Appendix A Eq. (A.6)),} \\
 &= \int_{\partial\Omega} \mathbf{e}_v \cdot (\mathbf{E}_{\varepsilon} \mathbf{n}) \, dS = \langle \mathbf{e}_v, \mathbf{E}_{\varepsilon} \mathbf{n} \rangle_{\partial\Omega}.
 \end{aligned} \tag{4.15}$$

512 The imposition of the velocity field along the boundary $\mathbf{e}_v = \partial_t \mathbf{u}$ corresponds to a Dirichlet
 513 condition. Setting $\mathbf{E}_{\varepsilon} \mathbf{n} = \boldsymbol{\Sigma} \mathbf{n}$ (the traction) corresponds to a Neumann condition. Consider

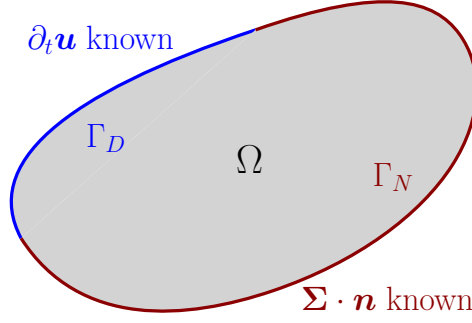


Figure 4.1: A 2D continuum with Neumann and Dirichlet boundary conditions

514 a partition of the boundary $\partial\Omega = \bar{\Gamma}_N \cup \bar{\Gamma}_D$ and $\Gamma_N \cap \Gamma_D = \{\emptyset\}$, where a Dirichlet and a
 515 Neumann condition applies on the open subset Γ_D and Γ_N respectively (see Fig. 4.1). Then
 516 the final pH formulation reads

$$\begin{aligned}
 \frac{\partial}{\partial t} \begin{pmatrix} \alpha_v \\ \mathbf{A}_\varepsilon \end{pmatrix} &= \underbrace{\begin{bmatrix} \mathbf{0} & \text{Div} \\ \text{Grad} & \mathbf{0} \end{bmatrix}}_{\mathcal{J}} \begin{pmatrix} \mathbf{e}_v \\ \mathbf{E}_\varepsilon \end{pmatrix}, \\
 \mathbf{u}_\partial &= \underbrace{\begin{bmatrix} \gamma_0^{\Gamma_D} & \mathbf{0} \\ \mathbf{0} & \gamma_n^{\Gamma_N} \end{bmatrix}}_{\mathcal{B}_\partial} \begin{pmatrix} \mathbf{e}_v \\ \mathbf{E}_\varepsilon \end{pmatrix}, \\
 \mathbf{y}_\partial &= \underbrace{\begin{bmatrix} \mathbf{0} & \gamma_n^{\Gamma_D} \\ \gamma_0^{\Gamma_N} & \mathbf{0} \end{bmatrix}}_{\mathcal{C}_\partial} \begin{pmatrix} \mathbf{e}_v \\ \mathbf{E}_\varepsilon \end{pmatrix},
 \end{aligned} \tag{4.16}$$

517 where $\gamma_0^{\Gamma_*}$ denotes the trace over the set Γ_* , namely $\gamma_0^{\Gamma_*} \mathbf{e}_v = \mathbf{e}_v|_{\Gamma_*}$. Furthermore, $\gamma_n^{\Gamma_*}$ denotes
 518 the normal trace over the set Γ_* , namely $\gamma_n^{\Gamma_*} \mathbf{E}_\varepsilon = \mathbf{E}_\varepsilon \mathbf{n}|_{\Gamma_*}$.

Conjecture 1 (Stokes-Dirac structure for elastodynamics)

Let $H^{\text{Grad}}(\Omega, \mathbb{V})$ the space of vectors with symmetric gradient in $L^2(\Omega, \mathbb{S})$ and $H^{\text{Div}}(\Omega, \mathbb{S})$ denote the space of symmetric tensors with divergence in $L^2(\Omega, \mathbb{V})$. Consider the following definitions

$$\begin{aligned}
 H &:= H^{\text{Grad}}(\Omega, \mathbb{V}) \times H^{\text{Div}}(\Omega, \mathbb{S}), \\
 F &:= L^2(\Omega, \mathbb{V}) \times L^2(\Omega, \mathbb{S}), \\
 F_\partial &:= L^2(\Gamma_D, \mathbb{V}) \times L^2(\Gamma_N, \mathbb{V}).
 \end{aligned}$$

519 The set

$$D_{\mathcal{J}} = \left\{ \begin{pmatrix} \mathbf{f} \\ \mathbf{f}_\partial \\ \mathbf{e} \\ \mathbf{e}_\partial \end{pmatrix} \mid \mathbf{e} \in H, \mathbf{f} = -\mathcal{J}\mathbf{e}, \mathbf{f}_\partial = \mathcal{B}_\partial \mathbf{e}, \mathbf{e}_\partial = \mathcal{C}_\partial \mathbf{e} \right\}, \tag{4.17}$$

where $\mathbf{e} = (\mathbf{e}_v, \mathbf{E}_\varepsilon)$ and $\mathcal{J}, \mathcal{B}_\partial, \mathcal{C}_\partial$ are defined in (4.16), is a Stokes–Dirac structure with respect to the pairing

$$\langle\langle (\mathbf{f}^1, \mathbf{f}_\partial^1, \mathbf{e}^1, \mathbf{e}_\partial^1), (\mathbf{f}^2, \mathbf{f}_\partial^2, \mathbf{e}^2, \mathbf{e}_\partial^2) \rangle\rangle := \langle \mathbf{e}^1, \mathbf{f}^2 \rangle_F + \langle \mathbf{e}^2, \mathbf{f}^1 \rangle_F + \langle \mathbf{e}_\partial^1, \mathbf{f}_\partial^2 \rangle_{F_\partial} + \langle \mathbf{e}_\partial^2, \mathbf{f}_\partial^1 \rangle_{F_\partial}, \quad (4.18)$$

where

$$\langle\langle (\mathbf{a}, \mathbf{b}), (\mathbf{c}, \mathbf{d}) \rangle\rangle_{F_\partial} = \int_{\Gamma_D} \mathbf{a} \cdot \mathbf{c} \, dS + \int_{\Gamma_N} \mathbf{b} \cdot \mathbf{d} \, dS, \quad \mathbf{a}, \mathbf{b}, \mathbf{c}, \mathbf{d} \in \mathbb{V}.$$

Crucial points to obtain a rigorous proof The crucial point that needs to be elucidated is where the boundary variables live. These variables belong to the fractional Sobolev spaces $H^{\frac{1}{2}}(\partial\Omega, \mathbb{V})$, $H^{-\frac{1}{2}}(\partial\Omega, \mathbb{V})$ linked by duality with respect to the pivot space $L^2(\partial\Omega, \mathbb{V})$. This is why a L^2 inner product has been assumed as boundary inner product. Furthermore, the partition of the boundary due to the non uniform boundary control complicates the proof, since one has to properly connect the two partitions at their interconnection.

Elements to support the conjecture A Stokes–Dirac is characterized by the fact that $D_{\mathcal{J}} = D_{\mathcal{J}}^\perp$. Then one has to show that $D_{\mathcal{J}} \subset D_{\mathcal{J}}^\perp$ and $D_{\mathcal{J}}^\perp \subset D_{\mathcal{J}}$. The main steps of Theorem 3.6 in [LGZM05] are followed here to support the substantiation of the conjecture. The integration by parts formula is applied as in (4.15).

Step 1. To show that $D_{\mathcal{J}} \subset D_{\mathcal{J}}^\perp$, take $(\mathbf{f}, \mathbf{f}_\partial, \mathbf{e}, \mathbf{e}_\partial) \in D_{\mathcal{J}}$. Then

$$\begin{aligned} \langle\langle (\mathbf{f}, \mathbf{f}_\partial, \mathbf{e}, \mathbf{e}_\partial), (\mathbf{f}, \mathbf{f}_\partial, \mathbf{e}, \mathbf{e}_\partial) \rangle\rangle &= 2 \langle \mathbf{e}, \mathbf{f} \rangle_F + 2 \langle \mathbf{e}_\partial, \mathbf{f}_\partial \rangle_{F_\partial}, \\ &= 2 \langle \mathbf{e}, -\mathcal{J}\mathbf{e} \rangle_F + 2 \langle \mathbf{e}_\partial, \mathbf{f}_\partial \rangle_{F_\partial}, \\ &= -2 \int_{\Omega} \{ \mathbf{e}_v \cdot \text{Div } \mathbf{E}_\varepsilon + \mathbf{E}_\varepsilon : \text{Grad } \mathbf{e}_v \} \, d\Omega \\ &\quad + 2 \int_{\Gamma_D} \mathbf{e}_v \cdot (\mathbf{E}_\varepsilon \mathbf{n}) \, dS + 2 \int_{\Gamma_N} \mathbf{e}_v \cdot (\mathbf{E}_\varepsilon \mathbf{n}) \, dS, \\ &= -2 \int_{\Omega} \{ \mathbf{e}_v \cdot \text{Div } \mathbf{E}_\varepsilon + \mathbf{E}_\varepsilon : \text{Grad } \mathbf{e}_v \} \, d\Omega \\ &\quad + 2 \int_{\partial\Omega} \mathbf{e}_v \cdot (\mathbf{E}_\varepsilon \mathbf{n}) \, dS, = 0, \quad \text{from (4.15)}. \end{aligned}$$

This implies $D_{\mathcal{J}} \subset D_{\mathcal{J}}^\perp$.

Step 2. Take $(\phi, \phi_\partial, \epsilon, \epsilon_\partial) \in D_{\mathcal{J}}^\perp$ and $\mathbf{e}_0 \in H$ with compact support on Ω . This implies $\mathcal{B}_\partial \mathbf{e}_0 = (\mathbf{0}, \mathbf{0})$ and $\mathcal{C}_\partial \mathbf{e}_0 = (\mathbf{0}, \mathbf{0})$. Taking $(-\mathcal{J}\mathbf{e}_0, \mathbf{0}, \mathbf{e}_0, \mathbf{0}) \in D_{\mathcal{J}}$ then

$$\langle\langle (\phi, \phi_\partial, \epsilon, \epsilon_\partial), (\mathcal{J}\mathbf{e}_0, \mathbf{0}, \mathbf{e}_0, \mathbf{0}) \rangle\rangle = \langle \epsilon, -\mathcal{J}\mathbf{e}_0 \rangle_F + \langle \mathbf{e}_0, \phi \rangle_F = 0, \quad \forall \mathbf{e}_0 \in H.$$

It follows that $\epsilon \in H$ and $\phi = -\mathcal{J}\epsilon$.

Step 3. Take $(\phi, \phi_\partial, \epsilon, \epsilon_\partial) \in D_{\mathcal{J}}^\perp$ and $(f, f_\partial, e, e_\partial) \in D_{\mathcal{J}}$. Variables e, ϵ are indeed tuples containing a vector and a tensor, namely $e = (e_v, \mathbf{E}_\epsilon)$, $\epsilon = (\epsilon_v, \mathbf{E}_\epsilon)$. From step 2 and (4.18)

$$\begin{aligned} 0 &= -\langle e, \mathcal{J}\epsilon \rangle_F - \langle \mathcal{J}e, \epsilon \rangle_F + \langle e_\partial, \phi_\partial \rangle_{F_\partial} + \langle \epsilon_\partial, f_\partial \rangle_{F_\partial}, \\ &= -\int_{\partial\Omega} \{e_v \cdot (\mathbf{E}_\epsilon \mathbf{n}) + \epsilon_v \cdot (\mathbf{E}_\epsilon \mathbf{n})\} \, dS + \langle e_\partial, \phi_\partial \rangle_{F_\partial} + \langle \epsilon_\partial, f_\partial \rangle_{F_\partial} \end{aligned}$$

Consider the splitting of the boundary $\partial\Omega = \bar{\Gamma}_N \cup \bar{\Gamma}_D$

$$\begin{aligned} \int_{\partial\Omega} \{e_v \cdot (\mathbf{E}_\epsilon \cdot \mathbf{n}) + \epsilon_v \cdot (\mathbf{E}_\epsilon \cdot \mathbf{n})\} \, dS &= + \int_{\Gamma_N} \{e_{\partial,2} \cdot (\mathbf{E}_\epsilon \cdot \mathbf{n}) + \epsilon_v \cdot f_{\partial,2}\} \, dS, \\ &+ \int_{\Gamma_D} \{f_{\partial,1} \cdot (\mathbf{E}_\epsilon \cdot \mathbf{n}) + \epsilon_v \cdot e_{\partial,1}\} \, dS, \end{aligned}$$

where the elements of the vectors $f_\partial = (f_{\partial,1}, f_{\partial,2})$, $e_\partial = (e_{\partial,1}, e_{\partial,2})$ have been considered. By expanding of the terms $\langle e_\partial, \phi_\partial \rangle_{F_\partial} + \langle \epsilon_\partial, f_\partial \rangle_{F_\partial}$ and given the fact that e_∂, f_∂ have arbitrary values then

$$\phi_\partial = \begin{bmatrix} \gamma_0^{\Gamma_D} & \mathbf{0} \\ \mathbf{0} & \gamma_n^{\Gamma_N} \end{bmatrix} \begin{pmatrix} \epsilon_v \\ \mathbf{E}_\epsilon \end{pmatrix}, \quad \epsilon_\partial = \begin{bmatrix} \mathbf{0} & \gamma_n^{\Gamma_D} \\ \gamma_0^{\Gamma_N} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \epsilon_v \\ \mathbf{E}_\epsilon \end{pmatrix},$$

meaning that $D_{\mathcal{J}}^\perp \subset D_{\mathcal{J}}$.

Linear elasticity falls within the assumption of [Skr19]. Therefore, it is a well posed boundary control pH system. A question that naturally arises is how to reformulate this system using the language of differential geometry. This is possible through the usage of vector-valued differential forms. The interested reader may consult [Bre08].

4.3 Conclusion

In this chapter, the pH formulation of elasticity have been obtained. This model represents a generalization of the wave equation to higher dimensional variables. This leads to the introduction of symmetric tensorial quantities describing the state of stress and deformation within the body.

For a plane continuum with moderate thickness, it is possible to reduce the general three-dimensional mode to two uncoupled systems: one representing the in plane behavior ruled by 2D elasticity and one representing the out-of-plane deflection. This will be the object of the next chapter dedicated to the study of a pH formulation of plate bending. It is important to remember that plate models are just particular cases of three-dimensional elasticity.

Port-Hamiltonian plate theory

You get tragedy where the tree, instead of bending, breaks.

Culture and Value
Ludwig Wittgenstein

Contents

5.1	First order plate theory	34
------------	---	-----------

5.1.1	Mindlin-Reissner model	35
-------	----------------------------------	----

5.1.2	Kirchhoff-Love model	36
-------	--------------------------------	----

5.2	Port-Hamiltonian formulation of isotropic plates	38
------------	---	-----------

5.2.1	Port-Hamiltonian Mindlin plate	39
-------	--	----

5.2.2	Port-Hamiltonian Kirchhoff plate	43
-------	--	----

5.3	Laminated anisotropic plates	48
------------	---	-----------

5.3.1	Port-Hamiltonian laminated Mindlin plate	50
-------	--	----

5.3.2	Port-Hamiltonian laminated Kirchhoff plate	51
-------	--	----

5.4	Conclusion	52
------------	-----------------------------	-----------



lates are plane structural elements with a small thickness compared to the planar dimension. Thanks to this feature, it is not necessary to model plate structures using three-dimensional elasticity. Dimensional reduction strategies are employed to describe plate structures as two-dimensional problems. These strategies rely on an educated guess of the displacement field. For beams and plates this field is expressed in terms of unknown functions $\phi_i^j(x, y, t)$ that solely depends on the midplane coordinates (x, y)

$$u_i(x, y, z, t) = \sum_{j=0}^m (z)^j \phi_i^j(x, y, t).$$

where u_i , $i = \{x, y, z\}$ are the components of the displacement field. A first-order approximation is commonly used, meaning that a linear dependence on z is considered. Two main models arise from such a framework:

- the Mindlin-Reissner model for thick plates;

-
- the Kirchhoff-Love model for thin plates.

In this chapter it is shown how to formulate first-order plate models as pHs.

5.1 First order plate theory

As previously stated, first order theories assume a linear dependence on the vertical coordinate (cf. [Red06])

$$u_i(x, y, z, t) = \phi_i^0(x, y, t) + z\phi_i^1(x, y, t).$$

This hypothesis implies that the fibers, i.e. segments perpendicular to the mid-plane before deformation, remain straight after deformation. Additionally, for plate with moderate thickness the fibers are considered inextensible, meaning that $\phi_z^1 = 0$. These assumptions lead to the following displacement field

$$\begin{aligned} u_x(x, y, z, t) &= u_x^0(x, y, t) - z\theta_x(x, y, t), \\ u_y(x, y, z, t) &= u_y^0(x, y, t) - z\theta_y(x, y, t), \\ u_z(x, y, z, t) &= u_z^0(x, y, t), \end{aligned} \quad (5.1)$$

where $u_i(x, y, t) = \phi_i^0(x, y, t)$, $\theta_i(x, y, t) = -\phi_i^1(x, y, t)$. Assuming a linear elastic behavior, the 3D strain tensor for such a displacement field takes the form

$$\varepsilon_{\alpha\beta} = \frac{1}{2}(\partial_\beta u_\alpha + \partial_\alpha u_\beta) - z\frac{1}{2}(\partial_\beta \theta_\alpha + \partial_\alpha \theta_\beta) = \varepsilon_{\alpha\beta}^0 - z\kappa_{\alpha\beta}, \quad (5.2)$$

$$\varepsilon_{\alpha z} = \frac{1}{2}(\partial_\alpha u_z - \theta_\alpha) = \frac{1}{2}\gamma_\alpha, \quad (5.3)$$

where $\alpha = \{x, y\}$, $\beta = \{x, y\}$. The tensors ε^0 , κ , γ are called membrane, bending (or curvature) and shear strain tensor

$$\varepsilon^0 = \text{Grad } \mathbf{u}^0, \quad (5.4)$$

$$\kappa = \text{Grad } \boldsymbol{\theta}, \quad (5.5)$$

$$\gamma = \text{grad } u_z - \boldsymbol{\theta}. \quad (5.6)$$

where $\mathbf{u}^0 = (u_x, u_y)^\top$, $\boldsymbol{\theta} = (\theta_x, \theta_y)^\top$. For now, it is assumed that the material is isotropic, linear elastic (in Section §5.3 this hypothesis is removed). Recall the Hooke's law for 3D continua (see Eq. (4.5))

$$\boldsymbol{\Sigma} = \frac{E}{1+\nu} \left[\boldsymbol{\varepsilon} + \frac{\nu}{1-2\nu} \text{Tr}(\boldsymbol{\varepsilon}) \mathbf{I}_{3 \times 3} \right].$$

where E , ν are the Young modulus and Poisson ratio. The hypothesis of inextensible fibers implies $\varepsilon_{zz} = 0$. However, imposing a plane strain condition provides a model that is too stiff. Rather than a plain strain assumption, a plain stress hypothesis is used to derive the constitutive law for plates. The displacement field (5.1) is left unchanged, but, instead of ε_{zz} ,

Σ_{zz} is set to zero. If $\Sigma_{zz} = 0$, one gets

$$\varepsilon_{zz} = -\frac{\nu}{1-\nu}(\varepsilon_{xx} + \varepsilon_{yy}).$$

Consequently, it is computed

$$\text{Tr}(\boldsymbol{\varepsilon}) = \frac{1-2\nu}{1-\nu}(\varepsilon_{xx} + \varepsilon_{yy}).$$

The constitutive law for the in-plane stress takes the form

$$\boldsymbol{\Sigma}_{2D} = \boldsymbol{\mathcal{D}}_{2D} \boldsymbol{\varepsilon}_{2D},$$

584 where $\boldsymbol{\Sigma}_{2D} = \Sigma_{\alpha\beta}$, $\boldsymbol{\varepsilon}_{2D} = \varepsilon_{\alpha\beta}$ and

$$\boldsymbol{\mathcal{D}}_{2D} = \frac{E}{1-\nu^2} [(1-\nu)(\cdot) + \nu \text{Tr}(\cdot) \mathbf{I}_{2 \times 2}]. \quad (5.7)$$

585 Concerning the shear deformation, the constitutive law reduces to

$$\boldsymbol{\sigma}_s = G\boldsymbol{\gamma}, \quad (5.8)$$

586 where $\boldsymbol{\sigma}_s := \boldsymbol{\Sigma}_{\alpha,3}$ and $G = \frac{E}{2(1+\nu)}$ is the shear modulus. In the following sections, the most
587 common plate models will be presented.

588 5.1.1 Mindlin-Reissner model

589 The Mindlin-Reissner model [Rei47, Min51] represents a first-order shear deformation theory
590 for describing the bending of plate. The in-plane midplane displacement are zero $\mathbf{u}^0(x, y) = \mathbf{0}$
591 for an isotropic plate that experiences only bending. Hence, the displacement field reduces to

$$\begin{aligned} u_x(x, y, z) &= -z\partial_x\theta_x, \\ u_y(x, y, z) &= -z\partial_y\theta_y, \\ u_z(x, y, z) &= u_z^0(x, y). \end{aligned} \quad (5.9)$$

In pure bending, the strain tensor is given by

$$\boldsymbol{\varepsilon}_b := \boldsymbol{\varepsilon}_{2D}(\mathbf{u}^0 = \mathbf{0}) = -z\boldsymbol{\kappa},$$

with $\boldsymbol{\kappa}$ given by (5.5). Consequently, the stress tensor reads

$$\boldsymbol{\Sigma}_b := \boldsymbol{\Sigma}_{2D}(\mathbf{u}^0 = \mathbf{0}) = -z\boldsymbol{\mathcal{D}}_{2D}\boldsymbol{\kappa},$$

592 where $\boldsymbol{\mathcal{D}}_{2D}$ is defined in Eq. (5.7).
593

594 The undeformed middle plane of the plate is denoted by Ω . The total domain of the

plate is the product $\Omega \times (-h/2, h/2)$, where h is the constant thickness. To effectively reduce the problem from three- to two-dimensional, the stresses have to be integrated along the fibers. Since the stress varies linearly across the thickness, the stress has to be multiplied by z before the integration to get a non null contribution. The resulting quantity is called bending momenta tensor and is given by

$$\mathbf{M} := - \int_{-h/2}^{h/2} z \boldsymbol{\Sigma}_b \, dz = \mathcal{D}_b \boldsymbol{\kappa}, \quad (5.10)$$

where

$$\mathcal{D}_b = D_b [(1 - \nu)(\cdot) + \nu \operatorname{Tr}(\cdot) \mathbf{I}_{2 \times 2}], \quad \text{where} \quad D_b = \frac{Eh^3}{12(1 - \nu^2)}. \quad (5.11)$$

The shear stress has to be integrated along the fibers as well. Given the excessive rigidity of the shear contribution, a correction factor $k = 5/6$ [Red06, Chapter 10] is introduced

$$\mathbf{q} = \int_{-h/2}^{h/2} k \boldsymbol{\sigma}_s \, dz = kGh\boldsymbol{\gamma}, \quad (5.12)$$

where $\boldsymbol{\gamma}$ is defined in Eq. (5.6). The equations of motion can be obtained using Hamilton's principle. It consists in minimizing the total Lagrangian, given by $L = E_{\text{def}} - E_{\text{kin}}$, where E_{def} , E_{kin} are the deformation and kinetic energy

$$E_{\text{def}} = \frac{1}{2} \int_{\Omega} \int_{-h/2}^{h/2} \boldsymbol{\Sigma} : \boldsymbol{\varepsilon} \, d\Omega \, dz = \frac{1}{2} \int_{\Omega} \{ \mathbf{M} : \boldsymbol{\kappa} + \mathbf{q} \cdot \boldsymbol{\gamma} \} \, d\Omega, \quad (5.13)$$

$$E_{\text{kin}} = \frac{1}{2} \int_{\Omega} \int_{-h/2}^{h/2} \rho \|\partial_t \mathbf{u}\|^2 \, d\Omega \, dz = \frac{1}{2} \int_{\Omega} \left\{ \frac{\rho h^3}{12} \|\partial_t \boldsymbol{\theta}\|^2 + \rho h (\partial_t u_z)^2 \right\} \, d\Omega, \quad (5.14)$$

where ρ is the mass density. The Hamilton principle states that

$$\int_0^T \delta L \, dt = \int_0^T \{ \delta E_{\text{def}} - \delta E_{\text{kin}} \} \, dt = 0.$$

The final result is the following system of PDEs (for the detailed computations see [Red06, Chapter 10])

$$\begin{aligned} \rho h \frac{\partial^2 u_z}{\partial t^2} &= \operatorname{div} \mathbf{q}, & (x, y) \in \Omega, \\ \frac{\rho h^3}{12} \frac{\partial^2 \boldsymbol{\theta}}{\partial t^2} &= \operatorname{Div} \mathbf{M} + \mathbf{q}, \end{aligned} \quad (5.15)$$

with $\mathbf{M} = \mathcal{D}_b \operatorname{Grad} \boldsymbol{\theta}$ and $\mathbf{q} = kGh(\operatorname{grad} u_z - \boldsymbol{\theta})$. This PDE goes together with specified boundary conditions. Those will be detailed in 5.2.1.

5.1.2 Kirchhoff-Love model

The Kirchhoff model was formulated around 1850 and it is referred to as classical plate theory. The hypotheses on the displacement field consist of the following three points (see Fig. 5.1):

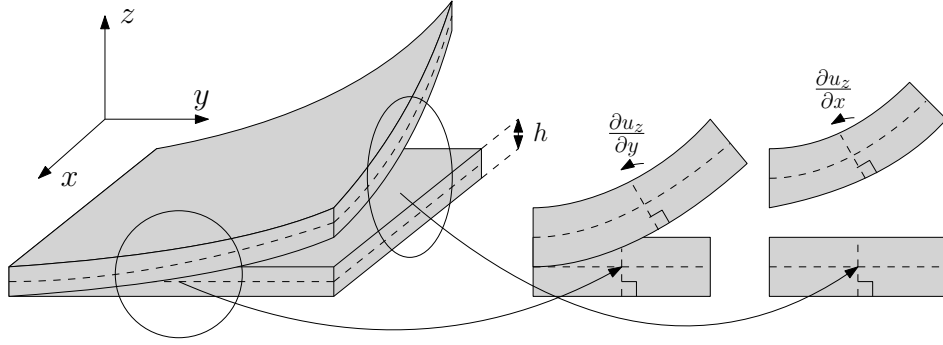


Figure 5.1: Kinematic assumption for the Kirchhoff plate

1. The fibers, segments perpendicular to the mid-plane before deformation, remain straight after deformation.
2. The fibers are inextensible.
3. While rotating, fibers remain perpendicular to the middle surface after deformation.

While the first two points are valid also for the Mindlin plate, the third assumption is specific to the Kirchhoff-Love model. Such an approximation is valid for plates having span-to-thickness ratio of the order of $L/h \approx 100 - 1000$ and implies zero transverse shear deformation

$$\gamma = 0 \implies \varepsilon_{xz} = -\theta_x + \frac{\partial u_z}{\partial x} = 0, \quad \varepsilon_{yz} = -\theta_y + \frac{\partial u_z}{\partial y} = 0.$$

The rotation vector is then related to the vertical displacement $\boldsymbol{\theta} = \text{grad } u_z$. Plugging this into (5.5), it is found

$$\boldsymbol{\kappa} = \text{Grad grad } u_z = \text{Hess } u_z. \quad (5.16)$$

Since the focus is on bending behavior, the in-plane displacement of the mid-plane are assumed to be zero $\mathbf{u}^0(x, y) = \mathbf{0}$. Hence, the displacement field assumes the form

$$\begin{aligned} u_x(x, y, z) &= -z \partial_x u_z, \\ u_y(x, y, z) &= -z \partial_y u_z, \\ u_z(x, y, z) &= u_z^0(x, y). \end{aligned} \quad (5.17)$$

For the Kirchhoff plate, the same link between the momenta and bending tensor holds

$$\mathbf{M} = \mathcal{D}_b \boldsymbol{\kappa},$$

where \mathcal{D}_b and $\boldsymbol{\kappa}$ are given in (5.11), (5.16) respectively. The equations of motion can be obtained using Hamilton's principle [Red06, Chapter 2]. The deformation energy, kinetic

energy and external work read

$$E_{\text{def}} = \frac{1}{2} \int_{\Omega} \int_{-h/2}^{h/2} \boldsymbol{\Sigma} : \boldsymbol{\varepsilon} \, d\Omega \, dz = \frac{1}{2} \int_{\Omega} \{ \mathbf{M} : \boldsymbol{\kappa} \} \, d\Omega, \quad (5.18)$$

$$E_{\text{kin}} = \frac{1}{2} \int_{\Omega} \int_{-h/2}^{h/2} \rho \, \|\partial_t \mathbf{u}\|^2 \, d\Omega \, dz \approx \frac{1}{2} \int_{\Omega} \rho h (\partial_t u_z)^2 \, d\Omega. \quad (5.19)$$

Remark 5 (Rotational energy)

For the kinetic energy the rotational contribution

$$E_{\text{rot}} = \frac{1}{2} \int_{\Omega} \int_{-h/2}^{h/2} \left\{ \rho (\partial_t u_x)^2 + (\partial_t u_y)^2 \right\} \, d\Omega \, dz = \frac{h^3}{24} \int_{\Omega} \rho \left\{ (\partial_{tx} u_z)^2 + (\partial_{ty} u_z)^2 \right\} \, d\Omega = O(h^3),$$

is neglected given the small thickness assumption.

The final result from the Hamilton's principle is the following PDE (for the detailed computations the reader may consult [Red06, Chapter 3])

$$\rho h \frac{\partial^2 u_z}{\partial t^2} = -\operatorname{div} \operatorname{Div}(\mathcal{D}_b \operatorname{Grad} \operatorname{grad} u_z), \quad (x, y) \in \Omega. \quad (5.20)$$

Developing the calculations, one obtains

$$\rho h \frac{\partial^2 u_z}{\partial t^2} = -D_b \Delta^2 u_z, \quad (x, y) \in \Omega,$$

where $\Delta^2 = \frac{\partial^4}{\partial x^4} + 2 \frac{\partial^2}{\partial x^2} \frac{\partial^2}{\partial y^2} + \frac{\partial^4}{\partial y^4}$ is the bi-Laplacian. Appropriate boundary conditions for this problem will be detailed in 5.2.2.

5.2 Port-Hamiltonian formulation of isotropic plates

In this section the pH formulation of the isotropic Mindlin and Kirchhoff plate models is detailed. In [MMB05], the Mindlin plate model was put in pH form by appropriate selection of the energy variables. However, the final system does not consider the nature of the different variables that come into play, leading to a non intrinsic final formulation. Additionally, this model was presented using the jet bundle formalism in [SS17]. The Kirchhoff model was never explored in the pH framework and represents an original contribution of this thesis. The interested reader can find in [RZ18] a rigorous mathematical treatment of the biharmonic problem and its decomposition in 2D geometries, but only for the static case (the 3D case, that does not relate to plate bending, is treated in [DZ18]).

5.2.1 Port-Hamiltonian Mindlin plate

Let $w := u_z$ denote the vertical displacement of the plate. Consider a bounded, connected domain $\Omega \subset \mathbb{R}^2$ and the Hamiltonian (total energy)

$$H = \frac{1}{2} \int_{\Omega} \left\{ \rho h \left(\frac{\partial w}{\partial t} \right)^2 + \frac{\rho h^3}{12} \left\| \frac{\partial \boldsymbol{\theta}}{\partial t} \right\|^2 + \mathbf{M} : \boldsymbol{\kappa} + \mathbf{q} \cdot \boldsymbol{\gamma} \right\} d\Omega, \quad (5.21)$$

where \mathbf{M} , $\boldsymbol{\kappa}$, \mathbf{q} , $\boldsymbol{\gamma}$ are defined in Eqs. (5.10), (5.5), (5.12), (5.6) respectively. The choice of the energy variables is the same as in [MMB05] but here scalar-, vector- and tensor-valued variables are gathered together:

$$\begin{aligned} \alpha_w &= \rho h \frac{\partial w}{\partial t}, & \text{Linear momentum,} & & \alpha_{\theta} &= \frac{\rho h^3}{12} \frac{\partial \boldsymbol{\theta}}{\partial t}, & \text{Angular momentum,} \\ \mathbf{A}_{\kappa} &= \boldsymbol{\kappa}, & \text{Curvature tensor,} & & \boldsymbol{\alpha}_{\gamma} &= \boldsymbol{\gamma}. & \text{Shear deformation.} \end{aligned} \quad (5.22)$$

The energy is now a quadratic function of the energy variables

$$H = \frac{1}{2} \int_{\Omega} \left\{ \frac{1}{\rho h} \alpha_w^2 + \frac{12}{\rho h^3} \|\alpha_{\theta}\|^2 + (\mathcal{D}_b \mathbf{A}_{\kappa}) : \mathbf{A}_{\kappa} + (\mathcal{D}_s \boldsymbol{\alpha}_{\gamma}) \cdot \boldsymbol{\alpha}_{\gamma} \right\} d\Omega, \quad (5.23)$$

where $\mathcal{D}_s := Ghk \mathbf{I}_{2 \times 2}$ and G is the shear modulus k the correction factor. The co-energy variables are found by computing the variational derivative of the Hamiltonian:

$$\begin{aligned} e_w &:= \frac{\delta H}{\delta \alpha_w} = \frac{\partial w}{\partial t}, & \text{Linear velocity,} & & e_{\theta} &:= \frac{\delta H}{\delta \alpha_{\theta}} = \frac{\partial \boldsymbol{\theta}}{\partial t}, & \text{Angular velocity,} \\ \mathbf{E}_{\kappa} &:= \frac{\delta H}{\delta \mathbf{A}_{\kappa}} = \mathbf{M}, & \text{Momenta tensor,} & & \mathbf{e}_{\gamma} &:= \frac{\delta H}{\delta \boldsymbol{\alpha}_{\gamma}} = \mathbf{q} & \text{Shear stress.} \end{aligned} \quad (5.24)$$

Proposition 4

The variational derivative of the Hamiltonian with respect to the curvature tensor is the momenta tensor $\frac{\delta H}{\delta \mathbf{A}_{\kappa}} = \mathbf{M}$.

Proof. The proof is analogous to the one already detailed in Prop. 3 □

Once the variables are concatenated together, the pH system is expressed as follows

$$\frac{\partial}{\partial t} \begin{pmatrix} \alpha_w \\ \alpha_{\theta} \\ \mathbf{A}_{\kappa} \\ \boldsymbol{\alpha}_{\gamma} \end{pmatrix} = \begin{bmatrix} 0 & 0 & 0 & \text{div} \\ \mathbf{0} & \mathbf{0} & \text{Div} & \mathbf{I}_{2 \times 2} \\ \mathbf{0} & \text{Grad} & \mathbf{0} & \mathbf{0} \\ \text{grad} & -\mathbf{I}_{2 \times 2} & \mathbf{0} & \mathbf{0} \end{bmatrix} \begin{pmatrix} e_w \\ e_{\theta} \\ \mathbf{E}_{\kappa} \\ e_{\gamma} \end{pmatrix}. \quad (5.25)$$

The first two equations are equivalent to (5.15). The last two equations, like (4.14) for 3D elasticity, represent the fact the higher order derivatives commute. We shall now establish the total energy balance in terms of boundary variables as they will be part of the underlying

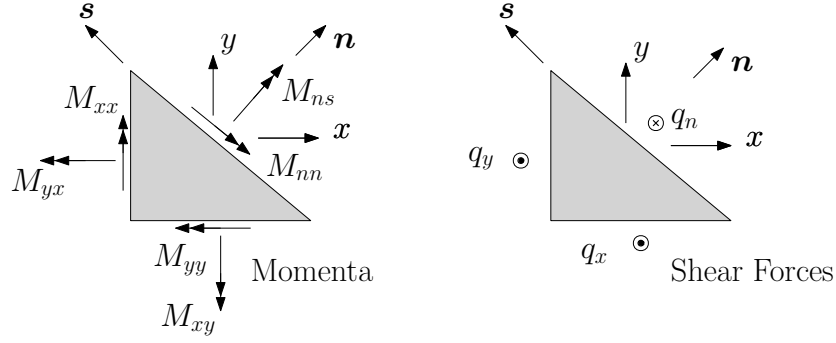


Figure 5.2: Cauchy law for momenta and forces at the boundary.

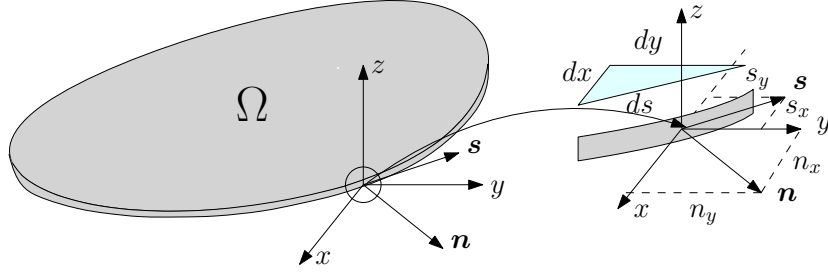


Figure 5.3: Reference frames and notations.

650 Stokes-Dirac structure of this model. The energy rate reads

$$\begin{aligned}
 \dot{H} &= \int_{\Omega} \left\{ \frac{\partial \alpha_w}{\partial t} e_w + \frac{\partial \alpha_\theta}{\partial t} \cdot \mathbf{e}_\theta + \frac{\partial \mathbf{A}_\kappa}{\partial t} : \mathbf{E}_\kappa + \frac{\partial \alpha_\gamma}{\partial t} \cdot \mathbf{e}_\gamma \right\} d\Omega \\
 &= \int_{\Omega} \{ \operatorname{div}(\mathbf{e}_\gamma) e_w + \operatorname{Div}(\mathbf{E}_\kappa) \cdot \mathbf{e}_\theta + \operatorname{Grad}(\mathbf{e}_\theta) : \mathbf{E}_\kappa + \operatorname{grad}(e_w) \cdot \mathbf{e}_\gamma \} d\Omega \quad \text{Stokes theorem,} \\
 &= \int_{\partial\Omega} \{ w_t q_n + \omega_n M_{nn} + \omega_s M_{ns} \} ds,
 \end{aligned} \tag{5.26}$$

651 where s is the curvilinear abscissa. The last integral is obtained by applying the Stokes
 652 theorem. The boundary variables appearing in the last line of (5.26) and illustrated in
 653 Fig. 5.2 are defined as follows:

$$\begin{aligned}
 \text{Shear force} \quad q_n &:= \mathbf{q} \cdot \mathbf{n} = \mathbf{e}_\gamma \cdot \mathbf{n}, \\
 \text{Flexural momentum} \quad M_{nn} &:= \mathbf{M} : (\mathbf{n} \otimes \mathbf{n}) = \mathbf{E}_\kappa : (\mathbf{n} \otimes \mathbf{n}), \\
 \text{Torsional momentum} \quad M_{ns} &:= \mathbf{M} : (\mathbf{s} \otimes \mathbf{n}) = \mathbf{E}_\kappa : (\mathbf{s} \otimes \mathbf{n}),
 \end{aligned} \tag{5.27}$$

654 Vectors \mathbf{n} and \mathbf{s} designate the normal and tangential unit vectors to the boundary, as shown
 655 in Fig. 5.3. Given two vectors $\mathbf{a} \in \mathbb{R}^n$, $\mathbf{a} \in \mathbb{R}^m$, the notation $\mathbf{a} \otimes \mathbf{b} = \mathbf{a} \mathbf{b}^\top \in \mathbb{R}^{n \times m}$ denotes the

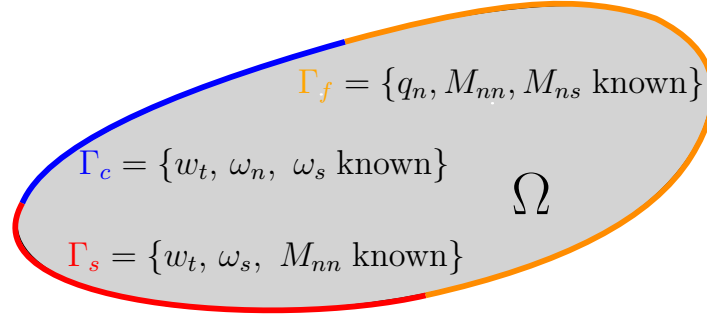


Figure 5.4: Boundary conditions for the Mindlin plate.

outer (or dyadic) product of two vectors. The corresponding power conjugated variables are

$$\begin{aligned}
 \text{Vertical velocity} \quad w_t &:= \frac{\partial w}{\partial t} = e_w, \\
 \text{Flexural rotation} \quad \omega_n &:= \frac{\partial \boldsymbol{\theta}}{\partial t} \cdot \mathbf{n} = \mathbf{e}_\theta \cdot \mathbf{n}, \\
 \text{Torsional rotation} \quad \omega_s &:= \frac{\partial \boldsymbol{\theta}}{\partial t} \cdot \mathbf{s} = \mathbf{e}_\theta \cdot \mathbf{s}.
 \end{aligned} \tag{5.28}$$

Consider a partition of the boundary $\partial\Omega = \bar{\Gamma}_C \cup \bar{\Gamma}_S \cup \bar{\Gamma}_F$, $\Gamma_C \cap \Gamma_S \cap \Gamma_F = \{\emptyset\}$. The open subset Γ_C , Γ_S , Γ_F could be empty. Given definitions (5.27), (5.28), the boundary conditions for the Mindlin plate [DHNLS99] (see Fig. 5.4) that are considered are:

- Clamped (C) on $\Gamma_C \subseteq \partial\Omega$: w_t , ω_n , ω_s known;
- Simply supported hard (S) on $\Gamma_S \subseteq \partial\Omega$: w_t , ω_s , M_{nn} known;
- Free (F) on $\Gamma_F \subseteq \partial\Omega$: M_{nn} , M_{ns} , q_n known.

Then the final pH formulation reads

$$\begin{aligned}
\frac{\partial}{\partial t} \begin{pmatrix} \alpha_w \\ \boldsymbol{\alpha}_\theta \\ \mathbf{A}_\kappa \\ \boldsymbol{\alpha}_\gamma \end{pmatrix} &= \underbrace{\begin{bmatrix} 0 & 0 & 0 & \text{div} \\ \mathbf{0} & \mathbf{0} & \text{Div} & \mathbf{I}_{2 \times 2} \\ \mathbf{0} & \text{Grad} & \mathbf{0} & \mathbf{0} \\ \text{grad} & -\mathbf{I}_{2 \times 2} & \mathbf{0} & \mathbf{0} \end{bmatrix}}_{\mathcal{J}} \begin{pmatrix} e_w \\ \mathbf{e}_\theta \\ \mathbf{E}_\kappa \\ e_\gamma \end{pmatrix}, \\
\mathbf{u}_\partial &= \underbrace{\begin{bmatrix} \gamma_0^{\Gamma_C} & 0 & 0 & 0 \\ 0 & \gamma_n^{\Gamma_C} & 0 & 0 \\ 0 & \gamma_s^{\Gamma_C} & 0 & 0 \\ \gamma_0^{\Gamma_S} & 0 & 0 & 0 \\ 0 & \gamma_s^{\Gamma_S} & 0 & 0 \\ 0 & 0 & \gamma_{nn}^{\Gamma_S} & 0 \\ 0 & 0 & \gamma_{nn}^{\Gamma_F} & 0 \\ 0 & 0 & \gamma_{ns}^{\Gamma_F} & 0 \\ 0 & 0 & 0 & \gamma_n^{\Gamma_F} \end{bmatrix}}_{\mathcal{B}_\partial} \begin{pmatrix} e_w \\ \mathbf{e}_\theta \\ \mathbf{E}_\kappa \\ e_\gamma \end{pmatrix}, \\
\mathbf{y}_\partial &= \underbrace{\begin{bmatrix} 0 & 0 & 0 & \gamma_n^{\Gamma_C} \\ 0 & 0 & \gamma_{nn}^{\Gamma_C} & 0 \\ 0 & 0 & \gamma_{ns}^{\Gamma_C} & 0 \\ 0 & 0 & 0 & \gamma_n^{\Gamma_S} \\ 0 & 0 & \gamma_{ns}^{\Gamma_S} & 0 \\ 0 & \gamma_n^{\Gamma_S} & 0 & 0 \\ 0 & \gamma_n^{\Gamma_F} & 0 & 0 \\ 0 & \gamma_s^{\Gamma_F} & 0 & 0 \\ \gamma_0^{\Gamma_F} & 0 & 0 & 0 \end{bmatrix}}_{\mathcal{C}_\partial} \begin{pmatrix} e_w \\ \mathbf{e}_\theta \\ \mathbf{E}_\kappa \\ e_\gamma \end{pmatrix},
\end{aligned} \tag{5.29}$$

664 where $\gamma_0^{\Gamma_*} a = a|_{\Gamma_*}$ denotes the trace over the set Γ_* . Furthermore, notations $\gamma_n^{\Gamma_*} \mathbf{a} = \mathbf{a} \cdot$
 665 $\mathbf{n}|_{\Gamma_*}$, $\gamma_s^{\Gamma_*} \mathbf{a} = \mathbf{a} \cdot \mathbf{s}|_{\Gamma_*}$ indicate the normal and tangential trace over the set Γ_* respectively.
 666 Symbols $\gamma_{nn}^{\Gamma_*}$, $\gamma_{ns}^{\Gamma_*}$ denote the normal-normal trace and the normal-tangential trace of tensor-
 667 valued functions, $\gamma_{nn}^{\Gamma_*} \mathbf{A} = \mathbf{A} : (\mathbf{n} \otimes \mathbf{n})|_{\Gamma_*}$, $\gamma_{ns}^{\Gamma_*} \mathbf{A} = \mathbf{A} : (\mathbf{n} \otimes \mathbf{s})|_{\Gamma_*}$.

Remark 6

669 It can be observed that the interconnection structure given by \mathcal{J} in (5.29) mimics that of the
 670 Timoshenko beam [JZ12, Chapter 7].

Conjecture 2 (Stokes-Dirac structure for the Mindlin plate)

Consider $\mathbb{V} = \mathbb{R}^2$, $\mathbb{S} = \mathbb{R}_{sym}^{2 \times 2}$ and let $H^1(\Omega)$ be the space of functions with gradient in $L^2(\Omega, \mathbb{V})$
 and $H^{\text{div}}(\Omega, \mathbb{V})$ the space of vector-valued functions with divergence in $L^2(\Omega)$. Furthermore,
 $H^1(\Omega, \mathbb{V})$ is the space of vectors with symmetric gradient in $L^2(\Omega, \mathbb{S})$ and $H^{\text{Div}}(\Omega, \mathbb{S})$ denote

the space of symmetric tensors with divergence in $L^2(\Omega, \mathbb{V})$. Consider the definitions

$$\begin{aligned} H &:= H^1(\Omega) \times H^{\text{Grad}}(\Omega, \mathbb{V}) \times H^{\text{Div}}(\Omega, \mathbb{S}) \times H^{\text{div}}(\Omega, \mathbb{V}), \\ F &:= L^2(\Omega) \times L^2(\Omega, \mathbb{V}) \times L^2(\Omega, \mathbb{S}) \times L^2(\Omega, \mathbb{V}), \\ F_\partial &:= L^2(\Gamma_C, \mathbb{R}^3) \times L^2(\Gamma_S, \mathbb{R}^3) \times L^2(\Gamma_F, \mathbb{R}^3). \end{aligned}$$

The set

$$D_{\mathcal{J}} = \left\{ \begin{pmatrix} \mathbf{f} \\ \mathbf{f}_\partial \\ \mathbf{e} \\ \mathbf{e}_\partial \end{pmatrix} \mid \mathbf{e} \in H, \mathbf{f} = -\mathcal{J}\mathbf{e}, \mathbf{f}_\partial = \mathcal{B}_\partial \mathbf{e}, \mathbf{e}_\partial = \mathcal{C}_\partial \mathbf{e} \right\}, \quad (5.30)$$

where $\mathbf{e} = (e_w, \mathbf{e}_\theta, \mathbf{E}_\kappa, \mathbf{e}_\gamma)$ and $\mathcal{J}, \mathcal{B}_\partial, \mathcal{C}_\partial$ are defined in (5.29), is a Stokes–Dirac structure with respect to the pairing

$$\langle \langle (\mathbf{f}^1, \mathbf{f}_\partial^1, \mathbf{e}^1, \mathbf{e}_\partial^1), (\mathbf{f}^2, \mathbf{f}_\partial^2, \mathbf{e}^2, \mathbf{e}_\partial^2) \rangle \rangle := \langle \mathbf{e}^1, \mathbf{f}^2 \rangle_F + \langle \mathbf{e}^2, \mathbf{f}^1 \rangle_F + \langle \mathbf{e}_\partial^1, \mathbf{f}_\partial^2 \rangle_{F_\partial} + \langle \mathbf{e}_\partial^2, \mathbf{f}_\partial^1 \rangle_{F_\partial}, \quad (5.31)$$

where $\mathbf{e}_\partial^i = (e_{\partial,1}^i, e_{\partial,2}^i, e_{\partial,3}^i)$, $\mathbf{f}_\partial^i = (f_{\partial,1}^i, f_{\partial,2}^i, f_{\partial,3}^i)$ and

$$\langle (\mathbf{a}, \mathbf{b}, \mathbf{c}), (\mathbf{d}, \mathbf{e}, \mathbf{f}) \rangle_{F_\partial} = \int_{\Gamma_C} \mathbf{a} \cdot \mathbf{d} \, dS + \int_{\Gamma_S} \mathbf{b} \cdot \mathbf{e} \, dS + \int_{\Gamma_F} \mathbf{c} \cdot \mathbf{f} \, dS, \quad \mathbf{a}, \mathbf{b}, \mathbf{c}, \mathbf{d}, \mathbf{e}, \mathbf{f} \in \mathbb{R}^3.$$

Crucial points and elements in favor of the conjecture Analogously to what was stated in Conjecture 1, the boundary spaces have to properly defined. If the integration by parts is carried out as in Eq. (5.26), one can follow the same lines of Conjecture 1 to support the present Conjecture.

The Mindlin plate falls within the assumption of [Skr19], hence it is a well posed boundary control pH systems.

5.2.2 Port-Hamiltonian Kirchhoff plate

Again the starting point is the Hamiltonian (total energy)

$$H = \frac{1}{2} \int_{\Omega} \left\{ \rho h \left(\frac{\partial w}{\partial t} \right)^2 + \mathbf{M} : \boldsymbol{\kappa} \right\} \, d\Omega, \quad (5.32)$$

where \mathbf{M} , $\boldsymbol{\kappa}$ are defined in Eqs. (5.10), (5.16). For what concerns the choice of the energy variables, a scalar and a tensor variable are considered:

$$\alpha_w = \rho h \frac{\partial w}{\partial t}, \quad \text{Linear momentum}, \quad \mathbf{A}_\kappa = \boldsymbol{\kappa}, \quad \text{Curvature tensor.} \quad (5.33)$$

684 The co-energy variables are found by computing the variational derivative of the Hamiltonian:

$$e_w := \frac{\delta H}{\delta \alpha_w} = \frac{\partial w}{\partial t}, \quad \text{Linear velocity}, \quad \mathbf{E}_\kappa := \frac{\delta H}{\delta \mathbf{A}_\kappa} = \mathbf{M}, \quad \text{Curvature tensor.} \quad (5.34)$$

685 The port-Hamiltonian system is then written as

$$\frac{\partial}{\partial t} \begin{pmatrix} \alpha_w \\ \mathbf{A}_\kappa \end{pmatrix} = \begin{bmatrix} 0 & -\text{div} \circ \text{Div} \\ \text{Grad} \circ \text{grad} & \mathbf{0} \end{bmatrix} \begin{pmatrix} e_w \\ \mathbf{E}_\kappa \end{pmatrix}. \quad (5.35)$$

The first equation is equivalent to (5.20). The last equation represent the fact the higher order derivatives commute

$$\begin{aligned} \partial_t \mathbf{A}_\kappa &= \text{Grad grad } e_w, \\ \partial_t \kappa &= \text{Grad grad } \partial_t w, \\ \partial_t \text{Grad grad } w &= \text{Grad grad } \partial_t w, \end{aligned}$$

686 The last equation holds for $w \in C^3(\Omega)$.

687 **Theorem 4**

688 *The operator $\text{Grad} \circ \text{grad}$, corresponding to the Hessian operator, is the adjoint of the double*
 689 *divergence $\text{div} \circ \text{Div}$.*

Proof. Let $\mathbb{S} = \mathbb{R}_{\text{sym}}^{d \times d}$ and consider the Hilbert space of the square integrable symmetric square tensors $L^2(\Omega, \mathbb{S})$ over an open connected set Ω (its inner product is defined in (4.12)). Consider the Hilbert space $L^2(\Omega)$ of scalar square integrable functions, endowed with the standard inner product. Consider the double divergence operator defined as:

$$\begin{aligned} \text{div Div} : L^2(\Omega, \mathbb{S}) &\rightarrow L^2(\Omega), \\ \Psi &\rightarrow \text{div Div } \Psi = \psi, \end{aligned} \quad \text{with } \psi = \text{div Div } \Psi = \sum_{i=1}^d \sum_{j=1}^d \frac{\partial^2 \Psi_{ij}}{\partial x_i \partial x_j}.$$

We shall identify div Div^*

$$\begin{aligned} \text{div Div}^* : L^2(\Omega) &\rightarrow L^2(\Omega, \mathbb{S}), \\ f &\rightarrow \text{div Div}^* f = \mathbf{F}, \end{aligned}$$

such that

$$\begin{aligned} \langle \text{div Div } \Psi, f \rangle_{L^2(\Omega)} &= \langle \Psi, \text{div Div}^* f \rangle_{L^2(\Omega, \mathbb{S})}, & \forall \Psi \in \text{Dom}(\text{div Div}) \subset L^2(\Omega, \mathbb{S}) \\ & & \forall f \in \text{Dom}(\text{div Div}^*) \subset L^2(\Omega) \end{aligned}$$

The function have to belong to the operator domain, so for instance $f \in C_0^2(\Omega) \in \text{Dom}(\text{div Div}^*)$ the space of twice differentiable scalar functions with compact support and Ψ can be chosen in the set $C_0^2(\Omega, \mathbb{S}) \in \text{Dom}(\text{div Div})$, the space of twice differentiable symmetric

tensors with compact support on Ω . A classical result is the fact that the adjoint of the vector divergence is $\operatorname{div}^* = -\operatorname{grad}$ as stated in [KZ15]. By theorem 3, it holds $\operatorname{Div}^* = -\operatorname{Grad}$. Considering that $\operatorname{div} \operatorname{Div} = \operatorname{div} \circ \operatorname{Div}$ is the composition of two different operators and that the adjoint of a composed operator is the adjoint of each operator in reverse order, i.e. $(B \circ C)^* = C^* \circ B^*$, then it can be stated

$$(\operatorname{div} \circ \operatorname{Div})^* = \operatorname{Div}^* \circ \operatorname{div}^* = \operatorname{Grad} \circ \operatorname{grad}.$$

690 Since only formal adjoints are being looked for, this concludes the proof. \square

691 The energy rate provides the boundary port variables

$$\begin{aligned} \dot{H} &= \int_{\Omega} \{ \partial_t \alpha_w e_w + \partial_t \mathbf{A}_{\kappa} : \mathbf{E}_{\kappa} \} \, d\Omega \\ &= \int_{\Omega} \{ -\operatorname{div} \operatorname{Div} \mathbf{E}_{\kappa} e_w + \operatorname{Grad} \operatorname{grad} e_w : \mathbf{E}_{\kappa} \} \, d\Omega, & \text{Stokes theorem} \\ &= \int_{\partial\Omega} \{ -\mathbf{n} \cdot \operatorname{Div} \mathbf{E}_{\kappa} e_w + (\mathbf{n} \otimes \operatorname{grad} e_w) : \mathbf{E}_{\kappa} \} \, ds, \\ &= \int_{\partial\Omega} \{ -\mathbf{n} \cdot \operatorname{Div} \mathbf{E}_{\kappa} e_w + \partial_{\mathbf{n}} e_w (\mathbf{n} \otimes \mathbf{n}) : \mathbf{E}_{\kappa} + \partial_{\mathbf{s}} e_w (\mathbf{n} \otimes \mathbf{s}) : \mathbf{E}_{\kappa} \} \, ds, & \text{Dyadic properties} \\ &= \int_{\partial\Omega} \{ \hat{q}_n w_t + \partial_{\mathbf{n}} w_t M_{nn} + \partial_{\mathbf{s}} w_t M_{ns} \} \, ds. \end{aligned} \tag{5.36}$$

692 where s is the curvilinear abscissa, $w_t := \partial_t w$ and $\partial_{\mathbf{s}} w_t$ denotes the directional derivative
693 along the tangential versor at the boundary. Additionally, the following definitions have been
694 introduced

$$\hat{q}_n := -\mathbf{n} \cdot \operatorname{Div}(\mathbf{E}_{\kappa}), \quad M_{nn} := (\mathbf{n} \otimes \mathbf{n}) : \mathbf{E}_{\kappa}, \quad M_{ns} := (\mathbf{n} \otimes \mathbf{s}) : \mathbf{E}_{\kappa}. \tag{5.37}$$

695 Variables w_t and $\partial_{\mathbf{s}} w_t$ are not independent as they are differentially related with respect to
696 derivation along \mathbf{s} (see for instance [TWK59, Chapter 4]). The tangential derivative has to be
697 moved on the torsional momentum M_{ns} . For sake of simplicity, $\partial\Omega$ is supposed to be regular.
698 Then the integration by parts provides

$$\int_{\partial\Omega} \partial_{\mathbf{s}} w_t M_{ns} \, ds = - \int_{\partial\Omega} \partial_{\mathbf{s}} M_{ns} w_t \, ds. \tag{5.38}$$

699 The final energy balance reads

$$\dot{H} = \int_{\partial\Omega} \{ w_t \tilde{q}_n + \partial_{\mathbf{n}} w_t M_{nn} \} \, ds, \tag{5.39}$$

700 where the boundary variables are

$$\begin{aligned} \text{Effective shear force} \quad \tilde{q}_n &:= \hat{q}_n - \partial_{\mathbf{s}} M_{ns}, \\ \text{Flexural momentum} \quad M_{nn} &:= \mathbf{M} : (\mathbf{n} \otimes \mathbf{n}) = \mathbf{E}_{\kappa} : (\mathbf{n} \otimes \mathbf{n}), \end{aligned} \tag{5.40}$$

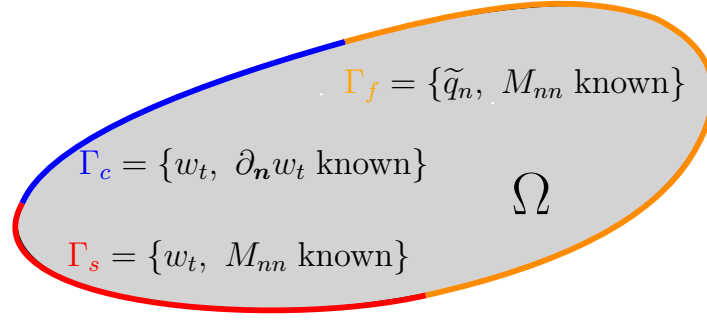


Figure 5.5: Boundary conditions for the Kirchhoff plate.

and \hat{q}_n is defined in (5.37). The corresponding power conjugated variables are:

$$\begin{aligned} \text{Vertical velocity} \quad w_t &:= \frac{\partial w}{\partial t} = e_w, \\ \text{Flexural rotation} \quad \partial_{\mathbf{n}} w_t &:= \nabla e_w \cdot \mathbf{n}. \end{aligned} \tag{5.41}$$

Consider a partition of the boundary $\partial\Omega = \bar{\Gamma}_C \cup \bar{\Gamma}_S \cup \bar{\Gamma}_F$, $\Gamma_C \cap \Gamma_S \cap \Gamma_F = \{\emptyset\}$, where $\Gamma_C, \Gamma_S, \Gamma_F$ are open subset of $\partial\Omega$. Given definitions (5.40), (5.41), the boundary conditions for the Kirchhoff plate [GSV18] are the following (see Fig. 5.5):

- Clamped (C) on $\Gamma_C \subseteq \partial\Omega$: $w_t, \partial_{\mathbf{n}} w_t$ known;
- Simply supported (S) on $\Gamma_S \subseteq \partial\Omega$: w_t, M_{nn} known;
- Free (F) on $\Gamma_F \subseteq \partial\Omega$: \tilde{q}_n, M_{nn} known.

Then the final pH formulation reads

$$\begin{aligned}
\frac{\partial}{\partial t} \begin{pmatrix} \alpha_w \\ \mathbf{A}_\kappa \end{pmatrix} &= \underbrace{\begin{bmatrix} 0 & -\operatorname{div} \circ \operatorname{Div} \\ \operatorname{Grad} \circ \operatorname{grad} & \mathbf{0} \end{bmatrix}}_{\mathcal{J}} \begin{pmatrix} e_w \\ \mathbf{E}_\kappa \end{pmatrix}, \\
\mathbf{u}_\partial &= \underbrace{\begin{bmatrix} \gamma_0^{\Gamma_C} & 0 \\ \gamma_1^{\Gamma_C} & 0 \\ \gamma_0^{\Gamma_S} & 0 \\ 0 & \gamma_{nn}^{\Gamma_S} \\ 0 & \gamma_{nn,1}^{\Gamma_F} \\ 0 & \gamma_{nn}^{\Gamma_F} \end{bmatrix}}_{\mathcal{B}_\partial} \begin{pmatrix} e_w \\ \mathbf{E}_\kappa \end{pmatrix}, \\
\mathbf{y}_\partial &= \underbrace{\begin{bmatrix} 0 & \gamma_{nn,1}^{\Gamma_C} \\ 0 & \gamma_{nn}^{\Gamma_C} \\ 0 & \gamma_{nn,1}^{\Gamma_S} \\ \gamma_1^{\Gamma_S} & 0 \\ \gamma_0^{\Gamma_F} & 0 \\ \gamma_1^{\Gamma_F} & 0 \end{bmatrix}}_{\mathcal{C}_\partial} \begin{pmatrix} e_w \\ \mathbf{E}_\kappa \end{pmatrix},
\end{aligned} \tag{5.42}$$

where $\gamma_0^{\Gamma_*} a = a|_{\Gamma_*}$ and $\gamma_1^{\Gamma_*} a = \partial_{\mathbf{n}} a|_{\Gamma_*}$ denote the standard and the normal derivative trace over the set Γ_* respectively. The symbol $\gamma_{nn,1}^{\Gamma_*}$ denotes the map $\gamma_{nn,1}^{\Gamma_*} \mathbf{A} = -\mathbf{n} \cdot \operatorname{Div} \mathbf{A} - \partial_s(\mathbf{A} : (\mathbf{n} \otimes \mathbf{s}))|_{\Gamma_*}$, while $\gamma_{nn}^{\Gamma_*} \mathbf{A} = \mathbf{A} : (\mathbf{n} \otimes \mathbf{n})|_{\Gamma_*}$ indicates the normal-normal trace of a tensor-valued function.

Remark 7

The interconnection structure \mathcal{J} in (5.42) mimics that of the Bernoulli beam [CRMPB17]. The double divergence and the Hessian coincide, in dimension one, with the second derivative.

Conjecture 3 (Stokes-Dirac structure for the Kirchhoff plate)

Consider $\mathbb{S} = \mathbb{R}_{\text{sym}}^{2 \times 2}$ and let $H^2(\Omega)$ be the space of functions with Hessian in $L^2(\Omega, \mathbb{S})$ and $H^{\operatorname{div} \operatorname{Div}}(\Omega, \mathbb{S})$ the space of vector-valued functions with double divergence in $L^2(\Omega)$. Consider the definitions

$$\begin{aligned}
H &:= H^2(\Omega) \times H^{\operatorname{div} \operatorname{Div}}(\Omega, \mathbb{S}), \\
F &:= L^2(\Omega) \times L^2(\Omega, \mathbb{S}), \\
F_\partial &:= L^2(\Gamma_C, \mathbb{R}^2) \times L^2(\Gamma_S, \mathbb{R}^2) \times L^2(\Gamma_F, \mathbb{R}^2).
\end{aligned}$$

The set

$$D_{\mathcal{J}} = \left\{ \begin{pmatrix} \mathbf{f} \\ \mathbf{f}_\partial \\ \mathbf{e} \\ \mathbf{e}_\partial \end{pmatrix} \mid \mathbf{e} \in H, \mathbf{f} = -\mathcal{J}\mathbf{e}, \mathbf{f}_\partial = \mathcal{B}_\partial \mathbf{e}, \mathbf{e}_\partial = \mathcal{C}_\partial \mathbf{e} \right\}, \tag{5.43}$$

where $\mathbf{e} = (e_w, \mathbf{E}_\kappa)$ and $\mathcal{J}, \mathcal{B}_\partial, \mathcal{C}_\partial$ are defined in (5.42), is a Stokes-Dirac structure with

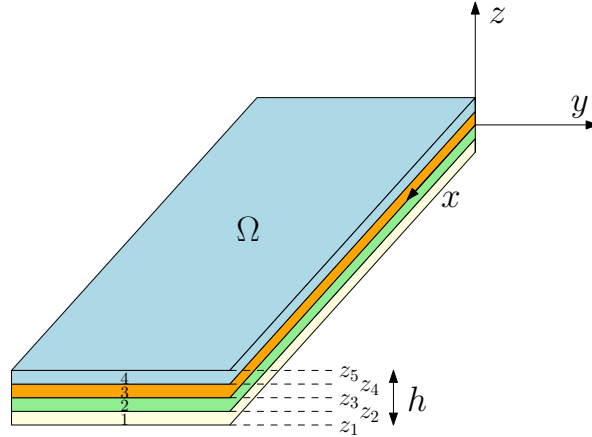


Figure 5.6: Laminated plate with 4 layers.

718 *respect to the pairing*

$$\langle\langle (\mathbf{f}^1, \mathbf{f}_{\partial}^1, \mathbf{e}^1, \mathbf{e}_{\partial}^1), (\mathbf{f}^2, \mathbf{f}_{\partial}^2, \mathbf{e}^2, \mathbf{e}_{\partial}^2) \rangle\rangle := \langle \mathbf{e}^1, \mathbf{f}^2 \rangle_F + \langle \mathbf{e}^2, \mathbf{f}^1 \rangle_F + \langle \mathbf{e}_{\partial}^1, \mathbf{f}_{\partial}^2 \rangle_{F_{\partial}} + \langle \mathbf{e}_{\partial}^2, \mathbf{f}_{\partial}^1 \rangle_{F_{\partial}}, \quad (5.44)$$

where $\mathbf{e}_{\partial}^i = (\mathbf{e}_{\partial,1}^i, \mathbf{e}_{\partial,2}^i)$, $\mathbf{f}_{\partial}^i = (\mathbf{f}_{\partial,1}^i, \mathbf{f}_{\partial,2}^i)$ and

$$\langle (\mathbf{a}, \mathbf{b}, \mathbf{c}), (\mathbf{d}, \mathbf{e}, \mathbf{f}) \rangle_{F_{\partial}} = \int_{\Gamma_C} \mathbf{a} \cdot \mathbf{d} \, dS + \int_{\Gamma_S} \mathbf{b} \cdot \mathbf{e} \, dS + \int_{\Gamma_F} \mathbf{c} \cdot \mathbf{f} \, dS, \quad \mathbf{a}, \mathbf{b}, \mathbf{c}, \mathbf{d}, \mathbf{e}, \mathbf{f} \in \mathbb{R}^2.$$

719 **Validity of the conjecture** *The integration by parts has to be carried as in Eq. (5.36) to*
 720 *retrieve a similar discussion to the one in Conjecture 1.*

721 5.3 Laminated anisotropic plates

722 Until now homogeneous isotropic materials have been considered. For this class of materials,
 723 the membrane and bending problems are decoupled. In aeronautical applications, structure
 724 are made up of laminae of different materials to enhance the mechanical properties of the
 725 resulting structure. In some cases, a certain coupling is desired, to increase the aerodynamical
 726 performance of the wing as it deforms.

727 Consider again the deformation field given by (5.1)

$$\begin{aligned} \mathbf{u}(x, y, z, t) &= \mathbf{u}^0(x, y, t) - z\boldsymbol{\theta}(x, y, t), \\ u_z(x, y, z, t) &= u_z^0(x, y, t), \end{aligned}$$

728 where $\mathbf{u} = (u_x, u_y)$. The link between in-plane deformation (5.2) and the membrane and

bending contribution (5.4), (5.5).

$$\varepsilon_{2D} = \varepsilon^0 - z\kappa \quad \text{where} \quad \varepsilon^0 = \text{Grad } \mathbf{u}^0, \quad \kappa = \text{Grad } \boldsymbol{\theta}. \quad (5.45)$$

Assume that each layer is an anisotropic material under plane stress condition. Then, it holds (see [Red03, Chapter 1] for details)

$$\boldsymbol{\Sigma}_{2D}^i = \mathcal{D}_{2D}^i \boldsymbol{\varepsilon}_{2D}^i,$$

where i indicates the layer under consideration. The matrix \mathcal{D}_{2D}^i depends on the properties of each material. To reduce the problem to bi-dimensional, the stresses have to be integrated along the thickness. Differently from isotropic plate, for laminated anisotropic plates the membrane and bending behavior are coupled. To see this consider the membrane and bending resultant of the stress

$$\mathbf{N} := \int_{-h/2}^{h/2} \boldsymbol{\Sigma}_{2D} \, dz, \quad \mathbf{M} := \int_{-h/2}^{h/2} -z \boldsymbol{\Sigma}_{2D} \, dz. \quad (5.46)$$

Since the stress are discontinuous due to the change of constitutive law along the thickness, the integration has to be performed lamina-wise. Once the computations are carried out, it is found

$$\begin{pmatrix} \mathbf{N} \\ \mathbf{M} \end{pmatrix} = \begin{bmatrix} \mathcal{D}_m & \mathcal{D}_c \\ \mathcal{D}_c & \mathcal{D}_b \end{bmatrix} \begin{pmatrix} \varepsilon^0 \\ \kappa \end{pmatrix}, \quad (5.47)$$

where

$$\mathcal{D}_m = \sum_{i=1}^{n_{\text{layer}}} \mathcal{D}_{2D}^i (z_{i+1} - z_i), \quad \mathcal{D}_c = -\frac{1}{2} \sum_{i=1}^{n_{\text{layer}}} \mathcal{D}_{2D}^i (z_{i+1}^2 - z_i^2), \quad \mathcal{D}_b = \frac{1}{3} \sum_{i=1}^{n_{\text{layer}}} \mathcal{D}_{2D}^i (z_{i+1}^3 - z_i^3), \quad (5.48)$$

and n_{layer} is the number of layers and z_i represents the height of the i^{th} layer (see Fig. 5.6). The coupling term \mathcal{D}_c disappears if a symmetric configuration is considered. For the shear contribution it is obtained

$$\mathbf{q} := \int_{-h/2}^{h/2} \boldsymbol{\sigma}_s \, dz = \mathcal{D}_s \boldsymbol{\gamma}, \quad \text{where} \quad \boldsymbol{\gamma} = \text{grad } u_z - \boldsymbol{\theta}. \quad (5.49)$$

The tensor \mathcal{D}_s is not diagonal as in the isotropic case, cf. §5.2.1.

In the following section it is shown how anisotropic laminated plates can be formulated as pHs.

5.3.1 Port-Hamiltonian laminated Mindlin plate

For a shear deformable laminated plate the kinetic and deformation energy read

$$E_{\text{kin}} = \frac{1}{2} \int_{\Omega} \left\{ \rho h \left\| \frac{\partial \mathbf{u}^0}{\partial t} \right\|^2 + \rho h \left(\frac{\partial u_z}{\partial t} \right)^2 + \frac{\rho h^3}{12} \left\| \frac{\partial \boldsymbol{\theta}}{\partial t} \right\|^2 \right\} d\Omega,$$

$$E_{\text{def}} = \frac{1}{2} \int_{\Omega} \left\{ \mathbf{N} : \boldsymbol{\varepsilon}^0 + \mathbf{M} : \boldsymbol{\kappa} + \mathbf{q} \cdot \boldsymbol{\gamma} \right\} d\Omega.$$

By using Hamilton's principle the equations of motion are retrieved (see [Red03, Chapter 3] for an exhaustive explanation)

$$\begin{aligned} \rho h \frac{\partial^2 \mathbf{u}^0}{\partial t^2} &= \text{Div } \mathbf{N}, \\ \rho h \frac{\partial^2 u_z}{\partial t^2} &= \text{div } \mathbf{q}, \\ \frac{\rho h^3}{12} \frac{\partial^2 \boldsymbol{\theta}}{\partial t^2} &= \text{Div } \mathbf{M} + \mathbf{q}, \end{aligned} \tag{5.50}$$

where \mathbf{N} , \mathbf{M} , \mathbf{q} are defined in Eqs. (5.47), (5.49). To get a port-Hamiltonian formulation, the following energy variable are chosen

$$\begin{aligned} \boldsymbol{\alpha}_u &= \rho h \frac{\partial \mathbf{u}^0}{\partial t}, & \alpha_w &= \rho h \frac{\partial u_z}{\partial t}, & \boldsymbol{\alpha}_\theta &= \frac{\rho h^3}{12} \frac{\partial \boldsymbol{\theta}}{\partial t}, \\ \mathbf{A}_{\varepsilon^0} &= \boldsymbol{\varepsilon}^0, & \mathbf{A}_\kappa &= \boldsymbol{\kappa}, & \boldsymbol{\alpha}_\gamma &= \boldsymbol{\gamma}. \end{aligned} \tag{5.51}$$

This choice highlights the nature of the problem in which the membrane part (equivalent to a 2D elasticity problem) and the bending part interact. The total energy $H = E_{\text{kin}} + E_{\text{def}}$ is now a quadratic function of the energy variables

$$E_{\text{kin}} = \frac{1}{2} \int_{\Omega} \left\{ \frac{1}{\rho h} \left\| \frac{\partial \boldsymbol{\alpha}_u}{\partial t} \right\|^2 + \frac{1}{\rho h} \left(\frac{\partial \alpha_w}{\partial t} \right)^2 + \frac{12}{\rho h^3} \left\| \frac{\partial \boldsymbol{\alpha}_\theta}{\partial t} \right\|^2 \right\} d\Omega,$$

$$E_{\text{def}} = \frac{1}{2} \int_{\Omega} \left\{ (\mathcal{D}_m \mathbf{A}_{\varepsilon^0} + \mathcal{D}_c \mathbf{A}_\kappa) : \mathbf{A}_{\varepsilon^0} + (\mathcal{D}_c \mathbf{A}_{\varepsilon^0} + \mathcal{D}_b \mathbf{A}_\kappa) : \mathbf{A}_\kappa + (\mathcal{D}_s \boldsymbol{\alpha}_\gamma) \cdot \boldsymbol{\alpha}_\gamma \right\} d\Omega,$$

The co-energies are equal to

$$\begin{aligned} e_w &:= \frac{\delta H}{\delta \alpha_w} = \frac{\partial u_z}{\partial t}, & e_\theta &:= \frac{\delta H}{\delta \boldsymbol{\alpha}_\theta} = \frac{\partial \boldsymbol{\theta}}{\partial t}, \\ \mathbf{E}_\kappa &:= \frac{\delta H}{\delta \mathbf{A}_\kappa} = \mathbf{M}, & e_\gamma &:= \frac{\delta H}{\delta \boldsymbol{\alpha}_\gamma} = \mathbf{q} \end{aligned} \tag{5.52}$$

754 The final pH formulation is found as usual considering the dynamics (5.50) and fact that
 755 higher derivatives commute

$$\frac{\partial}{\partial t} \begin{pmatrix} \alpha_u \\ \alpha_w \\ \alpha_\theta \\ \mathbf{A}_{\varepsilon^0} \\ \mathbf{A}_\kappa \\ \alpha_\gamma \end{pmatrix} = \begin{bmatrix} \mathbf{0} & \mathbf{0} & \mathbf{0} & \text{Div} & \mathbf{0} & \mathbf{0} \\ 0 & 0 & 0 & 0 & 0 & \text{div} \\ \mathbf{0} & \mathbf{0} & \mathbf{0} & \mathbf{0} & \text{Div} & \mathbf{I}_{2 \times 2} \\ \text{Grad} & \mathbf{0} & \mathbf{0} & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \text{Grad} & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \text{grad} & -\mathbf{I}_{2 \times 2} & \mathbf{0} & \mathbf{0} & \mathbf{0} \end{bmatrix} \begin{pmatrix} e_u \\ e_w \\ e_\theta \\ \mathbf{E}_{\varepsilon^0} \\ \mathbf{E}_\kappa \\ e_\gamma \end{pmatrix}. \quad (5.53)$$

756 The coupling between the membrane and bending part is clear when considering the link
 757 between energy and co-energy variables

$$\begin{pmatrix} e_u \\ e_w \\ e_\theta \\ \mathbf{E}_{\varepsilon^0} \\ \mathbf{E}_\kappa \\ e_\gamma \end{pmatrix} = \begin{bmatrix} \frac{1}{\rho h} \mathbf{I}_{2 \times 2} & \mathbf{0} & \mathbf{0} & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ 0 & \frac{1}{\rho h} & 0 & 0 & 0 & 0 \\ \mathbf{0} & \mathbf{0} & \frac{12}{\rho h^3} \mathbf{I}_{2 \times 2} & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{0} & \mathcal{D}_m & \mathcal{D}_c & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{0} & \mathcal{D}_c & \mathcal{D}_b & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{0} & \mathbf{0} & \mathbf{0} & \mathcal{D}_s \end{bmatrix} \begin{pmatrix} \alpha_u \\ \alpha_w \\ \alpha_\theta \\ \mathbf{A}_{\varepsilon^0} \\ \mathbf{A}_\kappa \\ \alpha_\gamma \end{pmatrix}. \quad (5.54)$$

758 Again appropriate boundary variables and a suitable Stokes-Dirac structure can be found for
 759 this model. The final formulation is just a superposition of systems (4.16) and (5.29).

760 5.3.2 Port-Hamiltonian laminated Kirchhoff plate

According to the Kirchhoff hypotheses the kinetic and deformation energies reduce to

$$E_{\text{kin}} = \frac{1}{2} \int_{\Omega} \left\{ \rho h \left\| \frac{\partial \mathbf{u}^0}{\partial t} \right\|^2 + \rho h \left(\frac{\partial u_z}{\partial t} \right)^2 \right\} d\Omega,$$

$$E_{\text{def}} = \frac{1}{2} \int_{\Omega} \left\{ \mathbf{N} : \varepsilon^0 + \mathbf{M} : \kappa \right\} d\Omega,$$

761 where κ is defined in Eq. (5.5). Furthermore, as stated in Remark 5, the rotational contri-
 762 bution in the kinetic energy has been neglected. The equations of motion are (see [Red03,
 763 Chapter 3] for an exhaustive explanation)

$$\rho h \frac{\partial^2 \mathbf{u}^0}{\partial t^2} = \text{Div } \mathbf{N},$$

$$\rho h \frac{\partial^2 u_z}{\partial t^2} = -\text{div Div } \mathbf{M},$$

(5.55)

where \mathbf{N} , \mathbf{M} are defined in Eqs. (5.47). To get a port-Hamiltonian formulation, the following energy variable are chosen

$$\begin{aligned}\alpha_u &= \rho h \frac{\partial \mathbf{u}^0}{\partial t}, & \alpha_w &= \rho h \frac{\partial u_z}{\partial t}, \\ \mathbf{A}_{\varepsilon^0} &= \boldsymbol{\varepsilon}^0, & \mathbf{A}_\kappa &= \boldsymbol{\kappa}.\end{aligned}\tag{5.56}$$

The total energy $H = E_{\text{kin}} + E_{\text{def}}$ is now a quadratic function of the energy variables

$$\begin{aligned}E_{\text{kin}} &= \frac{1}{2} \int_{\Omega} \left\{ \frac{1}{\rho h} \left\| \frac{\partial \alpha_u}{\partial t} \right\|^2 + \frac{1}{\rho h} \left(\frac{\partial \alpha_w}{\partial t} \right)^2 \right\} d\Omega, \\ E_{\text{def}} &= \frac{1}{2} \int_{\Omega} \{ (\mathcal{D}_m \mathbf{A}_{\varepsilon^0} + \mathcal{D}_c \mathbf{A}_\kappa) : \mathbf{A}_{\varepsilon^0} + (\mathcal{D}_c \mathbf{A}_{\varepsilon^0} + \mathcal{D}_b \mathbf{A}_\kappa) : \mathbf{A}_\kappa \} d\Omega,\end{aligned}$$

The co-energies are equal to

$$\begin{aligned}e_w &:= \frac{\delta H}{\delta \alpha_u} = \frac{\partial \mathbf{u}^0}{\partial t}, & e_w &:= \frac{\delta H}{\delta \alpha_w} = \frac{\partial u_z}{\partial t}, \\ \mathbf{E}_\kappa &:= \frac{\delta H}{\delta \mathbf{A}_{\varepsilon^0}} = \mathbf{N}, & \mathbf{E}_\kappa &:= \frac{\delta H}{\delta \mathbf{A}_\kappa} = \mathbf{M},\end{aligned}\tag{5.57}$$

The final pH formulation is found as usual considering the dynamics (5.55) and fact that higher derivatives commute

$$\frac{\partial}{\partial t} \begin{pmatrix} \alpha_u \\ \alpha_w \\ \mathbf{A}_{\varepsilon^0} \\ \mathbf{A}_\kappa \end{pmatrix} = \begin{bmatrix} \mathbf{0} & \mathbf{0} & \text{Div} & \mathbf{0} \\ 0 & 0 & 0 & -\text{div} \circ \text{Div} \\ \text{Grad} & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \text{Grad} \circ \text{grad} & \mathbf{0} & \mathbf{0} \end{bmatrix} \begin{pmatrix} e_u \\ e_w \\ \mathbf{E}_{\varepsilon^0} \\ \mathbf{E}_\kappa \end{pmatrix}.\tag{5.58}$$

Again, the coupling appears when considering the link between energy and co-energy variables

$$\begin{pmatrix} e_u \\ e_w \\ \mathbf{E}_{\varepsilon^0} \\ \mathbf{E}_\kappa \end{pmatrix} = \begin{bmatrix} \frac{1}{\rho h} \mathbf{I}_{2 \times 2} & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ 0 & \frac{1}{\rho h} & 0 & 0 \\ \mathbf{0} & \mathbf{0} & \mathcal{D}_m & \mathcal{D}_c \\ \mathbf{0} & \mathbf{0} & \mathcal{D}_c & \mathcal{D}_b \end{bmatrix} \begin{pmatrix} \alpha_u \\ \alpha_w \\ \mathbf{A}_{\varepsilon^0} \\ \mathbf{A}_\kappa \end{pmatrix}.\tag{5.59}$$

The energy rate provides the appropriate boundary conditions from which one can construct the Stokes-Dirac structure. The necessary computations are not performed here as the final result is just a juxtaposition of systems (4.16), (5.42).

5.4 Conclusion

In this chapter, a pH formulation for the most commonly used plate models has been detailed. Many open questions remain. In particular, how to generalize the results to shell problems, for which the domain is a surface embedded in the three dimensional space (a manifold). Computations get more involved in this case since the usage of differential geometry concepts

is unavoidable. These models are important since they are widely used in the aerospace industry and ubiquitous in nature.

The reformulation of plate models using the language of differential geometry is another open research topic. Indeed, while for the Mindlin plate it should be possible to use vector-valued forms to obtain an equivalent system, for the Kirchhoff plate the task appears more involved. An interesting reference that can provide some ideas in this direction is [Yao11].

Thermoelasticity in port-Hamiltonian form

Eh bien, mon ami, la terre sera un jour ce cadavre refroidi. Elle deviendra inhabitable et sera inhabitée comme la lune, qui depuis longtemps a perdu sa chaleur vitale.

Vingt mille lieues sous les mers
Jules Verne

Contents

6.1	Port-Hamiltonian linear coupled thermoelasticity	55
6.1.1	The heat equation as a pH descriptor system	56
6.1.2	Classical thermoelasticity	57
6.1.3	Thermoelasticity as two coupled pHs	59
6.2	Thermoelastic port-Hamiltonian bending	61
6.2.1	Thermoelastic Euler-Bernoulli beam	61
6.2.2	Thermoelastic Kirchhoff plate	63
6.3	Conclusion	65



Thermoelasticity is the study of deformable bodies undergoing thermal excitations. It is a clear example of a multiphysics phenomenon since the heat transfer and elastic vibrations within the body mutually interact. In this chapter, a linear model of thermoelasticity is obtained under the pH formalism. Each physics is described separately and the final system is obtained considering a power-preserving interconnection of two pHs.

6.1 Port-Hamiltonian linear coupled thermoelasticity

In this section, a pH formulation of heat transfer is first introduced. The classical model of thermoelasticity is then recalled. The same model is found by interconnecting the heat equation and the linear elastodynamics problem seen as pHs. It is shown that the interconnection

preserves a quadratic functional that plays the role of a fictitious energy. The resulting system is dissipative with respect to this functional. The construction makes use of the intrinsic modularity of pHs [KZvdSB10].

6.1.1 The heat equation as a pH descriptor system

Consider the heat equation in a bounded connected set $\Omega \subset \mathbb{R}^d$, $d = \{1, 2, 3\}$, describing the evolution of the temperature field $T(\mathbf{x}, t)$

$$\rho c_\epsilon \frac{\partial T}{\partial t} = k \Delta T + r_Q, \quad \mathbf{x} \in \Omega, \quad (6.1)$$

where ρ , c_ϵ , k , r_Q are the mass density, the specific heat density at constant strain, the thermal diffusivity and an heat source. Symbol Δ denotes the Laplacian in \mathbb{R}^d . The Dirichlet and Neumann condition of this problem are

$$\begin{aligned} T \text{ known on } \Gamma_D^T, & \quad \text{Dirichlet condition,} \\ -k \text{ grad } T \cdot \mathbf{n} \text{ known on } \Gamma_N^T, & \quad \text{Neumann condition,} \end{aligned}$$

where a partition of the boundary $\partial\Omega = \Gamma_D^T \cup \Gamma_N^T$ has been considered. This model can be put in pH form by means of a canonical interconnection structure. An algebraic relationship that describes the Fourier law has to be incorporated in the model (cf. [Kot19, Chapter 2]). Here, a differential-algebraic formulation is exploited to obtain the same system.

Let T_0 be a constant reference temperature (the introduction of this variables is instrumental for coupled thermoelasticity). The functional

$$H_T = \frac{1}{2} \int_{\Omega} \rho c_\epsilon T_0 \left(\frac{T - T_0}{T_0} \right)^2 d\Omega$$

has the physical dimension of an energy and represents a Lyapunov functional of this system. Even though it does not represent the internal energy, it has some important properties. Select as energy variable

$$\alpha_T := \rho c_\epsilon (T - T_0),$$

whose corresponding co-energy is

$$e_T := \frac{\delta H_T}{\delta \alpha_T} = \frac{\alpha_T}{\rho c_\epsilon T_0} = \frac{T - T_0}{T_0} =: \theta.$$

Introducing the heat flux $\mathbf{j}_Q := -k \text{ grad } T$ as additional variable, the heat equation (6.1) is

equivalently reformulated as

$$\begin{aligned} \begin{bmatrix} 1 & 0 \\ \mathbf{0} & \mathbf{0} \end{bmatrix} \frac{\partial}{\partial t} \begin{pmatrix} \alpha_T \\ \mathbf{j}_Q \end{pmatrix} &= \begin{bmatrix} 0 & -\operatorname{div} \\ -\operatorname{grad} & -(T_0 k)^{-1} \end{bmatrix} \begin{pmatrix} e_T \\ \mathbf{j}_Q \end{pmatrix} + \begin{bmatrix} 1 \\ \mathbf{0} \end{bmatrix} u_T, \\ y_T &= \begin{bmatrix} 1 & \mathbf{0} \end{bmatrix} \begin{pmatrix} e_T \\ \mathbf{j}_Q \end{pmatrix}. \end{aligned} \quad (6.2)$$

with $u_T := r_Q$ and y_T represents the corresponding power-conjugated variable. In matrix notation, it is obtained

$$\begin{aligned} \mathcal{E}_T \partial_t \boldsymbol{\alpha}_T &= (\mathcal{J}_T - \mathcal{R}_T) \mathbf{e}_T + \mathcal{B}_T u_T, \\ y_d &= \mathcal{B}_T^* \mathbf{e}_T \end{aligned} \quad (6.3)$$

where $\boldsymbol{\alpha}_T = (\alpha_T, \mathbf{j}_Q)$, $\mathbf{e}_T = (e_T, \mathbf{j}_Q)$ and

$$\mathcal{E}_T = \begin{bmatrix} 1 & 0 \\ \mathbf{0} & \mathbf{0} \end{bmatrix}, \quad \mathcal{J}_T = \begin{bmatrix} 0 & -\operatorname{div} \\ -\operatorname{grad} & \mathbf{0} \end{bmatrix}, \quad \mathcal{R}_T = \begin{bmatrix} 0 & 0 \\ \mathbf{0} & (T_0 k)^{-1} \end{bmatrix}, \quad \mathcal{B}_T = \begin{bmatrix} 1 \\ \mathbf{0} \end{bmatrix}.$$

The system is an example of pH descriptor system (cf. [BMXZ18] for the finite dimensional case). The Hamiltonian reads

$$H_T = \frac{1}{2} \int_{\Omega} \mathbf{e}_T \cdot \mathcal{E}_T \boldsymbol{\alpha}_T \, d\Omega. \quad (6.4)$$

The power rate is then deduced

$$\begin{aligned} \dot{H}_T &= \int_{\Omega} \mathbf{e}_T \cdot \mathcal{E}_T \partial_t \boldsymbol{\alpha}_T \, d\Omega, \\ &= \int_{\Omega} \mathbf{e}_T \cdot \{(\mathcal{J}_T - \mathcal{R}_T) \mathbf{e} + \mathcal{B}_T u_T\} \, d\Omega, \\ &= \int_{\Omega} u_T y_T \, d\Omega - \int_{\Omega} \left(e_T \operatorname{div} \mathbf{j}_Q + \mathbf{j}_Q \operatorname{grad} e_T + \frac{\|\mathbf{j}_Q\|^2}{k T_0} \right) \, d\Omega, \\ &\leq \int_{\Omega} u_T y_T \, d\Omega - \int_{\partial\Omega} e_T \mathbf{j}_Q \cdot \mathbf{n} \, dS. \end{aligned} \quad (6.5)$$

This choice of Hamiltonian allows retrieving the classical boundary conditions and leads to a dissipative system. Other formulations, based on an entropy or internal energy functionals, are possible for the heat equation [DMSB09, SHM19a]. These provide an accrescent or a lossless system. Unfortunately these formulations are non linear and their discretization is a difficult task [SHM19b].

6.1.2 Classical thermoelasticity

The derivation of the classical theory of thermoelasticity is not carried out here. The reader may consult in [HE09, Chapter 1] or [Abe12, Chapter 8] for a detailed discussion on this topic.

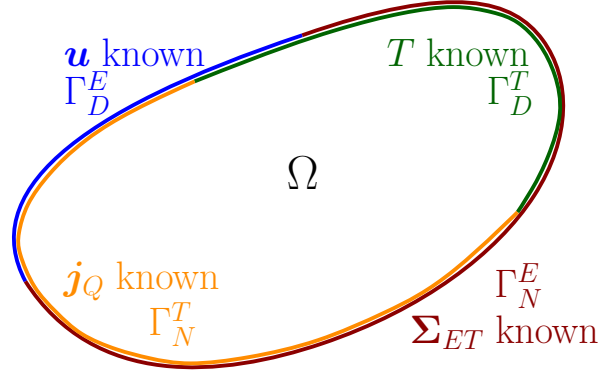


Figure 6.1: Boundary conditions for the thermoelastic problem.

840 Consider a bounded connected set $\Omega \subset \mathbb{R}^d$, $d = \{1, 2, 3\}$. The classical equations for linear
 841 fully-coupled thermoelasticity for an isotropic thermoelastic material are [Bio56, Car73]

$$\begin{aligned}
 \rho \frac{\partial^2 \mathbf{u}}{\partial t^2} &= \text{Div}(\boldsymbol{\Sigma}_{ET}), \\
 \rho c_\epsilon \frac{\partial T}{\partial t} &= -\text{div}(\mathbf{j}_Q) - \mathcal{C}_\beta : \frac{\partial \boldsymbol{\varepsilon}}{\partial t}, \\
 \boldsymbol{\Sigma}_{ET} &= \boldsymbol{\Sigma}_E + \boldsymbol{\Sigma}_T, \\
 \boldsymbol{\Sigma}_E &= 2\mu \boldsymbol{\varepsilon} + \lambda \text{Tr}(\boldsymbol{\varepsilon}) \mathbf{I}_{d \times d}, \\
 \boldsymbol{\Sigma}_T &= -\mathcal{C}_\beta \theta, \\
 \boldsymbol{\varepsilon} &= \text{Grad}(\mathbf{u}), \\
 \mathbf{j}_Q &= -k \text{grad } T.
 \end{aligned} \tag{6.6}$$

842 For simplicity the coupling term

$$\mathcal{C}_\beta := T_0 \beta (2\mu + d\lambda) \mathbf{I}_{d \times d}$$

843 has been introduced. Field \mathbf{u} is the displacement, $\boldsymbol{\varepsilon}$ is the infinitesimal strain tensor, $\boldsymbol{\Sigma}_E, \boldsymbol{\Sigma}_T$
 844 are the stress tensor contribution due to mechanical deformation and a thermal field. Co-
 845 efficients λ, μ are the Lamé parameters, and β the thermal expansion coefficient. Given a
 846 partition of the boundary $\partial\Omega = \Gamma_D^E \cup \Gamma_N^E = \Gamma_D^T \cup \Gamma_N^T$ for the elastic and thermal domain. The
 847 general boundary conditions read (see Fig. 6.1)

$$\begin{aligned}
 \mathbf{u} \text{ known on } \Gamma_D^E \times (0, +\infty), & \quad T \text{ known on } \Gamma_D^T \times (0, +\infty), \\
 \boldsymbol{\Sigma}_{ET} \cdot \mathbf{n} \text{ known on } \Gamma_N^E \times (0, +\infty), & \quad \mathbf{j}_Q \cdot \mathbf{n} \text{ known on } \Gamma_N^T \times (0, +\infty).
 \end{aligned} \tag{6.7}$$

848 In the following section an equivalent system is constructed by interconnecting the heat
 849 equation and the elastodynamics system in a structured manner.

6.1.3 Thermoelasticity as two coupled pHs

Consider again the equation of elasticity on $\Omega \subset \mathbb{R}^d$, $d = \{1, 2, 3\}$ (cf. Eq. (4.16)), together with a distributed input \mathbf{u}_E that plays the role of a distributed force

$$\begin{aligned} \frac{\partial}{\partial t} \begin{pmatrix} \boldsymbol{\alpha}_v \\ \mathbf{A}_\varepsilon \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & \text{Div} \\ \text{Grad} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_v \\ \mathbf{E}_\varepsilon \end{pmatrix} + \begin{bmatrix} \mathbf{I}_{d \times d} \\ \mathbf{0} \end{bmatrix} \mathbf{u}_E, \\ \mathbf{y}_E &= \begin{bmatrix} \mathbf{I}_{d \times d} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_v \\ \mathbf{E}_\varepsilon \end{pmatrix}, \end{aligned} \quad (6.8)$$

with Hamiltonian

$$H_E = \frac{1}{2} \int_{\Omega} \{ \boldsymbol{\alpha}_v \cdot \mathbf{e}_v + \mathbf{A}_\varepsilon : \mathbf{E}_\varepsilon \} \, d\Omega.$$

Recall the pH formulation of the heat equation (6.2)

$$\begin{aligned} \begin{bmatrix} 1 & \mathbf{0} \\ \mathbf{0} & \mathbf{0} \end{bmatrix} \frac{\partial}{\partial t} \begin{pmatrix} \alpha_T \\ \mathbf{j}_Q \end{pmatrix} &= \begin{bmatrix} 0 & -\text{div} \\ -\text{grad} & -(T_0 k)^{-1} \end{bmatrix} \begin{pmatrix} e_T \\ \mathbf{j}_Q \end{pmatrix} + \begin{bmatrix} 1 \\ \mathbf{0} \end{bmatrix} u_T, \\ \mathbf{y}_T &= \begin{bmatrix} 1 & \mathbf{0} \end{bmatrix} \begin{pmatrix} e_T \\ \mathbf{j}_Q \end{pmatrix}, \end{aligned} \quad (6.9)$$

with Hamiltonian H_T defined in (6.4). The linear thermoelastic problem can be expressed as a coupled port-Hamiltonian system. Consider the following interconnection

$$\mathbf{u}_E = -\text{Div}(\mathcal{C}_\beta \mathbf{y}_T), \quad u_T = -\mathcal{C}_\beta : \text{Grad}(\mathbf{y}_E). \quad (6.10)$$

The interconnection is power preserving as it can be compactly written as

$$\mathbf{u}_E = \mathcal{A}_\beta(\mathbf{y}_T), \quad u_T = -\mathcal{A}_\beta^*(\mathbf{y}_E).$$

where \mathcal{A}_β^* denotes the formal adjoint. The assertion is justified by the following proposition.

Proposition 5

Let $C_0^\infty(\Omega)$, $C_0^\infty(\Omega, \mathbb{R}^d)$ be the space of smooth functions and vector-valued functions respectively. Given $y_T \in C_0^\infty(\Omega)$, $\mathbf{y}_E \in C_0^\infty(\Omega, \mathbb{R}^d)$, the coupling operator

$$\begin{aligned} \mathcal{A}_\beta : C_0^\infty(\Omega) &\rightarrow C_0^\infty(\Omega, \mathbb{R}^d), \\ y_T &\rightarrow -\text{Div}(\mathcal{C}_\beta y_T) \end{aligned} \quad (6.11)$$

has formal adjoint

$$\begin{aligned} \mathcal{A}_\beta^* : C_0^\infty(\Omega, \mathbb{R}^d) &\rightarrow C_0^\infty(\Omega) \\ \mathbf{y}_E &\rightarrow -\mathcal{C}_\beta : \text{Grad}(\mathbf{y}_E) \end{aligned} \quad (6.12)$$

Proof. It is necessary to show

$$\langle \mathbf{y}_E, \mathcal{A}_\beta y_T \rangle_{L^2(\Omega, \mathbb{R}^d)} = \langle \mathcal{A}_\beta^* \mathbf{y}_E, y_T \rangle_{L^2(\Omega)}, \quad (6.13)$$

where for $\mathbf{u}_E, \mathbf{y}_E \in C_0^\infty(\Omega)$, $u_T, y_T \in C_0^\infty(\Omega)$

$$\langle \mathbf{u}_E, \mathbf{y}_E \rangle_{L^2(\Omega, \mathbb{R}^d)} = \int_{\Omega_E} \mathbf{u}_E \cdot \mathbf{y}_E \, d\Omega, \quad \langle u_T, y_T \rangle_{L^2(\Omega)} = \int_{\Omega_T} u_T y_T \, d\Omega. \quad (6.14)$$

The proof is a simple application of Th. 6

$$\begin{aligned} \langle \mathbf{y}_E, \mathcal{A}_\beta y_T \rangle_{L^2(\Omega, \mathbb{R}^d)} &= - \int_{\Omega} \mathbf{y}_E \cdot \text{Div}(\mathcal{C}_\beta y_T) \, d\Omega, \\ &= - \int_{\Omega} \text{Grad}(\mathbf{y}_E) : \mathcal{C}_\beta y_T \, d\Omega, \\ &= \int_{\Omega} \mathcal{A}_\beta^*(\mathbf{y}_E) y_T \, d\Omega, \\ &= \langle \mathcal{A}_\beta^* \mathbf{y}_E, y_T \rangle_{L^2(\Omega)}. \end{aligned} \quad (6.15)$$

This concludes the proof. \square

If the compact support assumption is removed, it is obtained

$$\begin{aligned} \langle u_T, y_T \rangle_{L^2(\Omega)} + \langle \mathbf{u}_E, \mathbf{y}_E \rangle_{L^2(\Omega, \mathbb{R}^3)} &= - \int_{\Omega} \{ (\mathcal{C}_\beta : \text{Grad} \, \mathbf{e}_v) e_T + \text{Div}(\mathcal{C}_\beta e_T) \cdot \mathbf{e}_v \} \, d\Omega, \\ &= - \int_{\Omega} \text{div}(e_T \mathcal{C}_\beta \cdot \mathbf{e}_v) \, d\Omega, \\ &= - \int_{\partial\Omega} (e_T \mathcal{C}_\beta \cdot \mathbf{n}) \cdot \mathbf{e}_v \, dS. \end{aligned} \quad (6.16)$$

Using the expression of y_T, \mathbf{y}_E , considering that T_0 is constant and applying Schwarz theorem for smooth function, the inputs are equal to

$$\mathbf{u}_E = \text{Div}(\boldsymbol{\Sigma}_T), \quad u_T = -\mathcal{C}_\beta : \text{Grad}(\mathbf{v}) = -\mathcal{C}_\beta : \frac{\partial \boldsymbol{\varepsilon}}{\partial t}.$$

The coupled thermoelastic problem can now be written as

$$\begin{bmatrix} \mathbf{1} & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \mathbf{1} & \mathbf{0} & \mathbf{0} \\ 0 & 0 & 1 & 0 \\ \mathbf{0} & \mathbf{0} & \mathbf{0} & \mathbf{0} \end{bmatrix} \frac{\partial}{\partial t} \begin{pmatrix} \boldsymbol{\alpha}_v \\ \mathbf{A}_\varepsilon \\ \alpha_T \\ \mathbf{j}_Q \end{pmatrix} = \begin{bmatrix} \mathbf{0} & \text{Div} & \mathcal{A}_\beta & \mathbf{0} \\ \text{Grad} & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ -\mathcal{A}_\beta^* & 0 & 0 & -\text{div} \\ \mathbf{0} & \mathbf{0} & -\text{grad} & -(T_0 k)^{-1} \end{bmatrix} \begin{pmatrix} \mathbf{e}_v \\ \mathbf{E}_\varepsilon \\ e_T \\ \mathbf{j}_Q \end{pmatrix}, \quad (6.17)$$

with total energy given by $H = H_E + H_T$. The power balance for each subsystem is given by

$$\dot{H}_E = \int_{\Omega} \mathbf{u}_E \cdot \mathbf{y}_E \, d\Omega + \int_{\partial\Omega} \mathbf{e}_v \cdot (\mathbf{E}_\varepsilon \cdot \mathbf{n}) \, dS, \quad (6.18)$$

$$\dot{H}_T \leq \int_{\Omega} u_T y_T \, d\Omega - \int_{\partial\Omega} \theta \mathbf{j}_Q \cdot \mathbf{n} \, dS, \quad (6.19)$$

The overall power balance is easily computed considering Eqs. (6.18) (6.19) and (6.16)

$$\dot{H} = \dot{H}_E + \dot{H}_T \leq \int_{\partial\Omega} \{[\mathbf{E}_\varepsilon - e_T \mathcal{C}_\beta] \cdot \mathbf{n}\} \cdot \mathbf{e}_v \, dS - \int_{\partial\Omega} \theta \, \mathbf{j}_Q \cdot \mathbf{n} \, dS. \quad (6.20)$$

From the power balance the classical boundary conditions are retrieved. This allows defining appropriate boundary operators for the thermoelastic problem

$$\mathbf{u}_\partial = \underbrace{\begin{bmatrix} \gamma_0^{\Gamma_D^E} & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \gamma_n^{\Gamma_N^E} & -\gamma_n^{\Gamma_N^E}(\mathcal{C}_\beta \cdot) & \mathbf{0} \\ 0 & 0 & \gamma_0^{\Gamma_D^T} & 0 \\ \mathbf{0} & \mathbf{0} & \mathbf{0} & \gamma_n^{\Gamma_N^T} \end{bmatrix}}_{\mathcal{B}_\partial} \begin{pmatrix} \mathbf{e}_v \\ \mathbf{E}_\varepsilon \\ e_T \\ \mathbf{j}_Q \end{pmatrix}, \quad \mathbf{y}_\partial = \underbrace{\begin{bmatrix} \mathbf{0} & \gamma_n^{\Gamma_D^E} & -\gamma_n^{\Gamma_D^E}(\mathcal{C}_\beta \cdot) & \mathbf{0} \\ \gamma_0^{\Gamma_N^E} & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{0} & \gamma_n^{\Gamma_D^T} \\ 0 & 0 & \gamma_0^{\Gamma_N^T} & 0 \end{bmatrix}}_{\mathcal{C}_\partial} \begin{pmatrix} \mathbf{e}_v \\ \mathbf{E}_\varepsilon \\ e_T \\ \mathbf{j}_Q \end{pmatrix}. \quad (6.21)$$

System (6.17) together with (6.21) is a pH system with boundary control and observation. Indeed, the classical thermoelastic problem can be modeled as two coupled systems, demonstrating the modularity of the pH paradigm.

6.2 Thermoelastic port-Hamiltonian bending

In this section, the thermoelastic bending of thin beam and plate structures is described as coupled interconnection of pHs. Starting from classical thermoelastic models and introducing a linear approximation of the temperature field along the thickness coordinate, a suitable pH formulation can be obtained.

6.2.1 Thermoelastic Euler-Bernoulli beam

The model for the linear thermoelastic vibrations of an isotropic thin rod is detailed in [Cha62, LR00]. The domain of the beam is uni-dimensional $\Omega_E = \{0, L\}$, while the thermal domain is three-dimensional $\Omega_T = \{0, L\} \times S$, where S is the set representing the beam cross section. The set S is assumed to be constant along the axis for simplicity. The ruling equations are

$$\begin{aligned} \rho A \frac{\partial^2 w}{\partial t^2} &= -EI \frac{\partial^4 w}{\partial x^4} - \beta E T_0 \frac{\partial^2}{\partial x^2} \int_S z \theta \, dx \, dy, & x \in \{0, L\} = \Omega_E, \\ \rho c_{\epsilon, B} T_0 \frac{\partial \theta}{\partial t} &= k T_0 \Delta \theta + \beta T_0 E z \frac{\partial^3 w}{\partial x^2 \partial t}, & (x, y, z) \in \Omega_E \times S = \Omega_T, \end{aligned} \quad (6.22)$$

where $w(x, t)$ is the vertical displacement of the beam $I = \int_S z^2 \, dx \, dy$ the second moment of area, E the Young modulus and A the cross section. The constant $c_{\epsilon, B}$ is due to the thermoelastic coupling (cf. [Cha62, LR00] for a detailed explanation). The other terms have meaning than in Section §6.1. Since the normalized temperature $\theta(x, y, z, t)$ depends on all spatial coordinates, the symbol $\Delta = \partial_{xx} + \partial_{yy} + \partial_{zz}$ is the Laplacian in three dimensions.

The physical constants are assumed to be constant for simplicity.

The coupling operator is defined as

$$\mathcal{A}_{\beta,B}(y_T) := -\beta ET_0 \partial_{xx} \left(\int_S z y_T \, dx \, dy \right). \quad (6.23)$$

To unveil an interconnection that is power with respect to a certain function, the formal adjoint of the coupling operator is needed.

Proposition 6

Let $C_0^\infty(\Omega_T)$, $C_0^\infty(\Omega_E)$ be the space of smooth functions with compact support defined on Ω_T and Ω_E respectively. Given $y_T \in C_0^\infty(\Omega_T)$, $y_E \in C_0^\infty(\Omega_E)$ the formal adjoint of the coupling operator is

$$\mathcal{A}_{\beta,B}^*(y_E) = -\beta ET_0 z \partial_{xx} y_E. \quad (6.24)$$

Proof. The formal adjoint is defined by the relation

$$\langle y_E, \mathcal{A}_{\beta,B} y_T \rangle_{L^2(\Omega_E)} = \langle \mathcal{A}_{\beta,B}^* y_E, y_T \rangle_{L^2(\Omega_T)}, \quad (6.25)$$

where for $u_E, y_E \in C_0^\infty(\Omega_E)$, $u_T, y_T \in C_0^\infty(\Omega_T)$

$$\langle u_E, y_E \rangle_{L^2(\Omega_E)} = \int_{\Omega_E} u_E y_E \, dx, \quad \langle u_T, y_T \rangle_{L^2(\Omega_T)} = \int_{\Omega_T} y_T y_T \, dx \, dy \, dz. \quad (6.26)$$

Using Def. (6.23) and the integration by parts, one finds

$$\begin{aligned} \langle y_E, \mathcal{A}_{\beta,B} y_T \rangle_{L^2(\Omega_E)} &= \int_{\Omega_E} y_E \mathcal{A}_{\beta,B} y_T \, dx, \\ &= - \int_{\Omega_E} y_E \beta ET_0 \partial_{xx} \left(\int_S z y_T \, dx \, dy \right) \, dx, \\ &= - \int_{\Omega_E} (\partial_{xx} y_E) \beta ET_0 \left(\int_S z y_T \, dx \, dy \right) \, dx, \end{aligned} \quad (6.27)$$

Since $\Omega_T = \Omega_E \times S$ and from the properties of multiple integrals, it is found

$$\begin{aligned} - \int_{\Omega_E} \partial_{xx} (y_E) \beta ET_0 \left(\int_S z y_T \, dx \, dy \right) \, dx &= - \int_{\Omega_E} \int_S (\partial_{xx} y_E) \beta ET_0 z y_T \, dx \, dx \, dy, \\ &= - \int_{\Omega_T} (\partial_{xx} y_E) \beta ET_0 z y_T \, dx \, dx \, dy, \\ &= \langle \mathcal{A}_{\beta,B}^* y_E, y_T \rangle_{L^2(\Omega_T)}. \end{aligned} \quad (6.28)$$

This concludes the proof. □

Using Eqs. (6.23) and (6.24), System (6.22), is rewritten as

$$\begin{aligned}\rho A \frac{\partial^2 w}{\partial t^2} &= -EI \frac{\partial^4 w}{\partial x^4} + \mathcal{A}_{\beta,B} \theta, \\ \rho c_{\epsilon,B} T_0 \frac{\partial \theta}{\partial t} &= k T_0 \Delta \theta - \mathcal{A}_{\beta,B}^* \frac{\partial w}{\partial t}.\end{aligned}\quad (6.29)$$

Consider the Hamiltonian functional

$$H = H_E + H_T = \frac{1}{2} \int_{\Omega_E} \left\{ \rho A \left(\frac{\partial w}{\partial t} \right)^2 + EI \left(\frac{\partial^2 w}{\partial x^2} \right)^2 \right\} dx + \frac{1}{2} \int_{\Omega_T} \rho c_{\epsilon,B} T_0 \theta^2 dx dy dz. \quad (6.30)$$

The energy variables are chosen to make the Hamiltonian functional quadratic

$$\alpha_w = \rho A \partial_t w, \quad \alpha_\kappa = \partial_{xx} w, \quad \alpha_T = \rho c_{\epsilon,B} T_0 \theta. \quad (6.31)$$

The corresponding co-energy variables evaluate to

$$e_w := \frac{\delta H}{\delta \alpha_w} = \partial_t w, \quad e_\kappa := \frac{\delta H}{\delta \alpha_\kappa} = EI \partial_{xx} w, \quad e_T := \frac{\delta H}{\delta \alpha_T} = \theta. \quad (6.32)$$

System (6.29) can now be rewritten as

$$\begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \frac{\partial}{\partial t} \begin{pmatrix} \alpha_w \\ \alpha_\kappa \\ \alpha_T \\ j_Q \end{pmatrix} = \begin{bmatrix} 0 & -\partial_{xx} & \mathcal{A}_{\beta,B} & 0 \\ \partial_{xx} & 0 & 0 & 0 \\ -\mathcal{A}_{\beta,B}^* & 0 & 0 & -\text{div} \\ 0 & 0 & -\text{grad} & -(kT_0)^{-1} \end{bmatrix} \begin{pmatrix} e_w \\ e_\kappa \\ e_T \\ j_Q \end{pmatrix}, \quad (6.33)$$

This system is the equivalent of (6.17) for bending of beams. Hence, following the same reasoning, it can be obtained starting from each subsystem in pH form by means of an appropriate interconnection.

6.2.2 Thermoelastic Kirchhoff plate

For the bending of thin plate, several different models have been proposed [Cha62, Lag89, Sim99, Nor06]. Here, the Chadwick model [Cha62] is considered. The thin plate occupies the open connected set $\Omega_E \times \left\{ -\frac{h}{2}, \frac{h}{2} \right\}$, where h is the plate thickness. The system of equations describe the midplane vertical displacement and the evolution of the temperature in the 3D domain

$$\begin{aligned}\rho h \frac{\partial^2 w}{\partial t^2} &= -D_b \Delta_{2D}^2 w - \frac{\beta T_0 E}{1-\nu} \Delta_{2D} \left(\int_{-h/2}^{h/2} z \theta dz \right), & (x, y) \in \Omega_E, \\ \rho c_{\epsilon,P} T_0 \frac{\partial \theta}{\partial t} &= -k T_0 \Delta_{3D} + \frac{\beta T_0 E z}{1-\nu} \Delta_{2D} \left(\frac{\partial w}{\partial t} \right), & (x, y, z) \in \Omega_E \times \left\{ -\frac{h}{2}, \frac{h}{2} \right\} = \Omega_T,\end{aligned}\quad (6.34)$$

where $w(x, y, t)$ is the vertical deflection, $D_b = \frac{E h^3}{12(1-\nu^2)}$ the bending rigidity (cf. Eq. (5.11)), ν the Poisson modulus and $c_{\epsilon,P}$ a constant (depending on the heat capacity at constant strain

and other coupling parameters, cf. [Cha62]). Symbols $\Delta_{2D} = \partial_{xx} + \partial_{yy}$, $\Delta_{3D} = \partial_{xx} + \partial_{yy} + \partial_{zz}$ are the two- and three-dimensional Laplacian.

The coupling operator is here defined as

$$\mathcal{A}_{\beta,P}(y_T) := -\frac{\beta T_0 E}{1-\nu} \Delta_{2D} \left(\int_{-h/2}^{h/2} z y_T \, dz \right). \quad (6.35)$$

Analogously with respect to the Euler-Bernoulli beam its formal adjoint is sought for.

Proposition 7

Let $C_0^\infty(\Omega_T)$, $C_0^\infty(\Omega_E)$ be the space of smooth functions with compact support defined on Ω_T and Ω_E respectively. Given $y_T \in C_0^\infty(\Omega_T)$, $y_E \in C_0^\infty(\Omega_E)$ the formal adjoint of the coupling operator is

$$\mathcal{A}_{\beta,B}^*(y_E) = -\frac{\beta T_0 E z}{1-\nu} \Delta_{2D} y_E. \quad (6.36)$$

Proof. The proof is completely identical to Prop. 6. □

System 6.34 is rewritten as

$$\begin{aligned} \rho h \frac{\partial^2 w}{\partial t^2} &= -D_b \Delta_{2D}^2 w + \mathcal{A}_{\beta,P} \theta, \\ \rho c_{\epsilon,P} T_0 \frac{\partial \theta}{\partial t} &= -k T_0 \Delta_{3D} \theta - \mathcal{A}_{\beta,P}^* \left(\frac{\partial w}{\partial t} \right), \end{aligned} \quad (6.37)$$

The Hamiltonian functional equals

$$\begin{aligned} H = H_E + H_T &= \frac{1}{2} \int_{\Omega_E} \left\{ \rho h \left(\frac{\partial w}{\partial t} \right)^2 + (\mathcal{D}_b \text{Hess}_{2D} w) : \text{Hess}_{2D} w \right\} \, dx \, dy \\ &+ \frac{1}{2} \int_{\Omega_T} \rho c_{\epsilon,P} T_0 \theta^2 \, dx \, dy \, dz, \end{aligned} \quad (6.38)$$

where Hess_{2D} is the Hessian in two dimensions and \mathcal{D}_b was defined in (5.11) (cf. Sec. §5.1.1).

The energy and co-energy variables are

$$\begin{aligned} \alpha_w &= \rho h \partial_t w, & \mathbf{A}_\kappa &= \text{Hess}_{2D} w, & \alpha_T &= \rho c_{\epsilon,P} T_0 \theta, \\ e_w &= \partial_t w, & \mathbf{E}_\kappa &= \mathcal{D}_b \text{Hess}_{2D} w, & e_T &= \theta. \end{aligned} \quad (6.39)$$

System (6.37) is rewritten as

$$\begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \frac{\partial}{\partial t} \begin{pmatrix} \alpha_w \\ \mathbf{A}_\kappa \\ \alpha_T \\ j_Q \end{pmatrix} = \begin{bmatrix} 0 & -\text{div Div}_{2D} & \mathcal{A}_{\beta,P} & 0 \\ \text{Hess}_{2D} & \mathbf{0} & \mathbf{0} & 0 \\ -\mathcal{A}_{\beta,P}^* & 0 & 0 & -\text{div}_{3D} \\ \mathbf{0} & \mathbf{0} & -\text{grad}_{3D} & -(kT_0)^{-1} \end{bmatrix} \begin{pmatrix} e_w \\ \mathbf{E}_\kappa \\ e_T \\ j_Q \end{pmatrix}, \quad (6.40)$$

The subscript $2D$, $3D$ refers to two- and three-dimensional operators respectively. The final system reproduces the same structured coupling already observed for (6.17), (6.33).

Remark 8

The thermoelastic bending of plates [AL00] and beams can be reduced to two problems defined on the same domain by introducing the following approximation of the temperature field

$$\theta(x, y, z) = \theta_0 + z\theta_1. \quad (6.41)$$

This is a reduction technique analogous to the one used to derive plate models.

6.3 Conclusion

In this chapter, it was shown how to derive linear thermoelastic problem as coupled pHs. This is especially interesting for the simulation of thermoelastic phenomena: each subsystem can be discretized separately and then coupled to the other using the discretized coupling operator.

To achieve a suitable formulation for the bending of plates and beams a linear approximation was introduced. However, if higher order theories are used for the bending behavior, the approximation of temperature field modifies accordingly, allowing for a better representation of temperature trend along the thickness.

949

Part III

950

Finite element structure preserving discretization

951

Partitioned finite element method

Every truth is simple... is that not doubly a lie?

Twilight of the Idols
Friedrich Nietzsche

Contents

7.1	Discretization under uniform boundary condition	69
7.1.1	General procedure	71
7.1.2	Linear case	80
7.1.3	Linear flexible structures	82
7.2	Mixed boundary conditions	91
7.2.1	Solution using Lagrange multipliers	92
7.2.2	Virtual domain decomposition	94
7.3	Conclusion	98



Discretization is the process of transferring continuous models into discrete counterparts. The discrete model should be faithful to the continuous one. To this aim, it is usually essential that the main properties of the continuous system are preserved at the discrete level. An algorithm that is capable of conserving properties at the discrete level is called structure-preserving [CMKO11]. In this chapter, a finite element method to spatially discretize infinite-dimensional pHs into finite-dimensional ones in a structure preserving manner is illustrated.

7.1 Discretization under uniform boundary condition

A discrete version of a infinite-dimensional pH system is meant to preserve the underlying properties related to power continuity. To achieve this purpose, the discretization procedure consists of two steps [KML18]:

- Finite-dimensional approximation of the Stokes-Dirac structure, i.e. the formally skew symmetric differential operator that defines the structure. The duality of the power

variables has to be mapped onto the finite approximation. The subspace of the discrete variables will be represented by a Dirac structure.

- The Hamiltonian requires as well a suitable discretization, which gives rise to a discrete Hamiltonian.

A structure-preserving discretization is able to construct an equivalent pH system that possess the structural properties of the original model:

Infinite dimensional pH system

PDE with distributed inputs:

$$\begin{aligned}\frac{\partial \alpha}{\partial t}(x, t) &= \mathcal{J} \frac{\delta H}{\delta \alpha} + \mathcal{B} \mathbf{u}(x, t), \\ \mathbf{y}(x, t) &= \mathcal{B}^* \frac{\delta H}{\delta \alpha}.\end{aligned}$$

Boundary conditions:

$$\mathbf{u}_\partial = \mathcal{B}_\partial \frac{\delta H}{\delta \alpha}, \quad \mathbf{y}_\partial = \mathcal{C}_\partial \frac{\delta H}{\delta \alpha}.$$

Power balance (Stokes Theorem):

$$\dot{H} = \int_{\partial\Omega} \mathbf{u}_\partial \cdot \mathbf{y}_\partial \, dS + \int_{\Omega} \mathbf{u} \cdot \mathbf{y} \, d\Omega.$$

Structure-preserving discretization

Resulting ODE:

$$\begin{aligned}\dot{\alpha}_d &= \mathbf{J} \nabla H_d + \mathbf{B}_d \mathbf{u}_d + \mathbf{B}_\partial \mathbf{u}_\partial, \\ \mathbf{y}_d &= \mathbf{B}_d^\top \nabla H_d, \\ \mathbf{y}_\partial &= \mathbf{B}_\partial^\top \nabla H_d.\end{aligned}$$

Discretized Hamiltonian:

$$H_d := H(\alpha \equiv \alpha_d).$$

Power balance:

$$\dot{H} = \mathbf{u}_\partial^\top \mathbf{y}_\partial + \mathbf{u}_d^\top \mathbf{y}_d.$$

In this thesis the partitioned finite element method (PFEM), originally presented in [CRML18, CRML19], is chosen to obtain discretized models of dpHs. This procedure boils down to three simple steps

1. The system is written in weak form;
2. An integration by parts is applied to highlight the appropriate boundary control;
3. A Galerkin method is employed to obtain a finite-dimensional system.

Once the system has been put into weak form, a subset of the equations is integrated by parts, so that boundary variables are naturally included into the formulation and appear as control inputs, the collocated outputs being defined accordingly. The discretization of energy and co-energy variables (and the associated test functions) leads directly to a full rank representation for the finite-dimensional pH system. This approach makes possible the usage of FEM software, like FEniCS [LMW⁺12], or Firedrake [RHM⁺17]. The procedure is universal, as it relies on a general integration by parts formula that characterizes multi-dimensional pHs. This is why the methodology is illustrated in all its generality and then detailed for some particular examples.

This methodology is easily applicable under a uniform causality assumption. The case of mixed boundary conditions requires additional care and will be treated in the subsequent Section §7.2.

7.1.1 General procedure

Given an open connected set $\Omega \in \mathbb{R}^d$, $d = \{1, 2, 3\}$, consider a generic pH system defined on Ω

$$\partial_t \boldsymbol{\alpha} = \mathcal{J} \mathbf{e}, \quad \boldsymbol{\alpha} \in X, \quad (7.1a)$$

$$\mathbf{e} := \delta_{\boldsymbol{\alpha}} H, \quad \mathbf{e} \in H^{\mathcal{J}}, \quad (7.1b)$$

$$\mathbf{u}_{\partial} = \mathcal{B}_{\partial} \mathbf{e}, \quad \mathbf{u}_{\partial} \in \mathbb{R}^m, \quad (7.1c)$$

$$\mathbf{y}_{\partial} = \mathcal{C}_{\partial} \mathbf{e}, \quad \mathbf{y}_{\partial} \in \mathbb{R}^m. \quad (7.1d)$$

The Hilbert space X , whose inner product is denoted by $\langle \cdot, \cdot \rangle_X$, is an appropriate Cartesian product of $L^2(\Omega)$ spaces which account for the nature of each variable (that can be scalar, vectorial or tensorial quantities). Its precise definition depends on the example upon consideration. For scalars $(a, b) \in L^2(\Omega)$, vectors $(\mathbf{a}, \mathbf{b}) \in L^2(\Omega, \mathbb{R}^d)$ and tensors $(\mathbf{A}, \mathbf{B}) \in L^2(\Omega, \mathbb{R}^{d \times d})$ the L^2 inner product is given by

$$\langle a, b \rangle_{L^2(\Omega)} = \int_{\Omega} ab \, d\Omega, \quad \langle \mathbf{a}, \mathbf{b} \rangle_{L^2(\Omega, \mathbb{R}^d)} = \int_{\Omega} \mathbf{a} \cdot \mathbf{b} \, d\Omega, \quad \langle \mathbf{A}, \mathbf{B} \rangle_{L^2(\Omega, \mathbb{R}^{d \times d})} = \int_{\Omega} \mathbf{A} : \mathbf{B} \, d\Omega. \quad (7.2)$$

For scalars $a_{\partial}, b_{\partial} \in L^2(\partial\Omega)$ and vectors $\mathbf{a}_{\partial}, \mathbf{b}_{\partial} \in L^2(\partial\Omega, \mathbb{R}^m)$ defined on the boundary the inner product is defined as

$$\langle a_{\partial}, b_{\partial} \rangle_{L^2(\partial\Omega)} = \int_{\partial\Omega} a_{\partial} b_{\partial} \, dS, \quad \langle \mathbf{a}_{\partial}, \mathbf{b}_{\partial} \rangle_{L^2(\partial\Omega, \mathbb{R}^m)} = \int_{\partial\Omega} \mathbf{a}_{\partial} \cdot \mathbf{b}_{\partial} \, dS. \quad (7.3)$$

For simplicity both $L^2(\partial\Omega)$ and $L^2(\partial\Omega, \mathbb{R}^m)$ are denoted by $L^2(\partial\Omega)$ in the inner product notation, since the use of bold letters clearly distinguishes vectors from scalars. The Hilbert space $H^{\mathcal{J}}$ is defined to be

$$H^{\mathcal{J}} := \{\mathbf{u} \in X \mid \mathcal{J} \mathbf{u} \in X\}. \quad (7.4)$$

The Hamiltonian functional of Eq. (7.1b) is allowed to be non linear in the energy variables

$$H = \int_{\Omega} \mathcal{H}(\boldsymbol{\alpha}) \, d\Omega,$$

where $\mathcal{H}(\boldsymbol{\alpha}) : X \rightarrow \mathbb{R}$ is a non linear function.

To applied this methodology the non linearities are restricted to the Hamiltonian and a uniform causality condition is supposed to characterize the system. It is required as well that the system admits a splitting of the variables. This requirement is always encounter in the following examples. These hypotheses are resumed in the following assumptions.

Assumption 1

Consider system (7.1a). It is assumed that the Hilbert space X admits the splitting $X = X_1 \times X_2$ (meaning that the system is made up of two main blocks). The operator \mathcal{J} is assumed to be skew-symmetric (or formally skew-adjoint) on X and linear:

$$\mathcal{J} = \mathcal{J}_a + \mathcal{J}_d, \quad (7.5)$$

where \mathcal{J}_a is the algebraic contribution (a skew-symmetric matrix) and \mathcal{J}_d the differential contribution. The algebraic part is assumed to take the form

$$\mathcal{J}_a = \begin{bmatrix} 0 & -\mathbf{L}^\top \\ \mathbf{L} & 0 \end{bmatrix}, \quad \begin{array}{l} \mathbf{L}^\top : X_2 \rightarrow X_1, \\ \mathbf{L} : X_1 \rightarrow X_2, \end{array} \quad (7.6)$$

where \mathbf{L} is a bounded operator. Analogously, the linear differential operator \mathcal{J}_d is assumed to be of the form

$$\mathcal{J}_d = \begin{bmatrix} 0 & -\mathcal{L}^* \\ \mathcal{L} & 0 \end{bmatrix}, \quad \begin{array}{l} \mathcal{L}^* : X_2 \rightarrow X_1, \\ \mathcal{L} : X_1 \rightarrow X_2, \end{array} \quad (7.7)$$

where \mathcal{L}^* denotes the formal adjoint of the linear differential operator \mathcal{L} . The operator \mathcal{L} is unbounded and can be either a first or a second order differential operator. In the latter case it can be expressed as $\mathcal{L} = \mathcal{L}_1 \circ \mathcal{L}_2$. Given the splitting $X_1 \times X_2 = X$ the Hilbert space $H^\mathcal{J}$ can be split as well as

$$H^\mathcal{J} = H^\mathcal{L} \times H^{-\mathcal{L}^*}, \quad \begin{array}{l} H^\mathcal{L} := \{\mathbf{u}_1 \in X_1 \mid \mathcal{L}\mathbf{u}_1 \in X_2\}, \\ H^{-\mathcal{L}^*} := \{\mathbf{u}_2 \in X_2 \mid -\mathcal{L}^*\mathbf{u}_2 \in X_1\} \end{array} \quad (7.8)$$

The boundary operators are then supposed to fulfill the following assumption, that guarantees a uniform causality condition.

Assumption 2

Assume that there exist two boundary operators $\mathcal{N}_{\partial,1}$, $\mathcal{N}_{\partial,2}$ such that for $(\mathbf{u}_1, \mathbf{u}_2) \in H^\mathcal{L} \times H^{-\mathcal{L}^*}$ a general integration by parts formula holds

$$\langle \mathbf{u}_2, \mathcal{L}\mathbf{u}_1 \rangle_{X_2} - \langle \mathcal{L}^*\mathbf{u}_2, \mathbf{u}_1 \rangle_{X_1} = \langle \mathcal{N}_{\partial,1}\mathbf{u}_1, \mathcal{N}_{\partial,2}\mathbf{u}_2 \rangle_{L^2(\partial\Omega)}. \quad (7.9)$$

The boundary operators $\mathcal{B}_\partial, \mathcal{C}_\partial$ of Eqs. (7.1c), (7.1d), are then assumed to verify, in an exclusive manner, either

$$\mathcal{B}_\partial = \begin{bmatrix} 0 & \mathcal{N}_{\partial,2} \end{bmatrix}, \quad \mathcal{C}_\partial = \begin{bmatrix} \mathcal{N}_{\partial,1} & 0 \end{bmatrix}, \quad (7.10)$$

or

$$\mathcal{B}_\partial = \begin{bmatrix} \mathcal{N}_{\partial,1} & 0 \end{bmatrix}, \quad \mathcal{C}_\partial = \begin{bmatrix} 0 & \mathcal{N}_{\partial,2} \end{bmatrix}. \quad (7.11)$$

Remark 9

The integration by part formula establishes a duality pairing between Sobolev spaces. This duality pairing is then compatible with an L^2 inner product in presence of a rigged Hilbert space

(Gelfand triple). Without entering into technical details, we shall always use this equivalence of representation. Therefore, the boundary integrals are expressed as L^2 inner product over the boundary.

Thanks to Assumption 1, System (7.1) is rewritten as

$$\partial_t \begin{pmatrix} \alpha_1 \\ \alpha_2 \end{pmatrix} = \begin{bmatrix} 0 & -\mathbf{L}^\top - \mathcal{L}^* \\ \mathbf{L} + \mathcal{L} & 0 \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix}, \quad \begin{matrix} \alpha_1 \in X_1, \\ \alpha_2 \in X_2, \end{matrix} \quad (7.12a)$$

$$\begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix} := \begin{pmatrix} \delta_{\alpha_1} H \\ \delta_{\alpha_2} H \end{pmatrix}, \quad \begin{matrix} \mathbf{e}_1 \in H^\mathcal{L}, \\ \mathbf{e}_2 \in H^{-\mathcal{L}*}. \end{matrix} \quad (7.12b)$$

Then thanks to Assumption 2 the boundary variables are given by

$$\mathbf{u}_\partial = \mathcal{N}_2 \mathbf{e}_2, \quad \mathbf{y}_\partial = \mathcal{N}_1 \mathbf{e}_1, \quad \mathbf{u}_\partial, \mathbf{y}_\partial \in \mathbb{R}^m, \quad (7.13)$$

if Eq. (7.10) holds. Otherwise, if Eq. (7.11) applies, then

$$\mathbf{u}_\partial = \mathcal{N}_1 \mathbf{e}_1, \quad \mathbf{y}_\partial = \mathcal{N}_2 \mathbf{e}_2, \quad \mathbf{u}_\partial, \mathbf{y}_\partial \in \mathbb{R}^m. \quad (7.14)$$

In both cases, the power balance reads

$$\begin{aligned} \dot{H} &= \langle \mathbf{e}_1, \partial_t \alpha_1 \rangle_{X_1} + \langle \mathbf{e}_2, \partial_t \alpha_2 \rangle_{X_2}, \\ &= \langle \mathbf{e}_1, -\mathcal{L}^* \mathbf{e}_2 \rangle_{X_1} + \langle \mathbf{e}_2, \mathcal{L} \mathbf{e}_1 \rangle_{X_2}, \\ &= \langle \mathcal{N}_{\partial,1} \mathbf{e}_1, \mathcal{N}_{\partial,2} \mathbf{e}_2 \rangle_{L^2(\partial\Omega)}, \\ &= \langle \mathbf{y}_\partial, \mathbf{u}_\partial \rangle_{L^2(\partial\Omega)}. \end{aligned} \quad (7.15)$$

We are now in a position to illustrate the methodology.

Step 1 First consider the weak form of system (7.12a), obtained by taking the L^2 inner product introducing an appropriate test function $\mathbf{v} = (\mathbf{v}_1, \mathbf{v}_2) \in X_1 \times X_2 = X$ and integrating over the domain Ω

$$\begin{aligned} \langle \mathbf{v}_1, \partial_t \alpha_1 \rangle_{X_1} &= -\langle \mathbf{v}_1, \mathbf{L}^\top \mathbf{e}_2 \rangle_{X_1} - \langle \mathbf{v}_1, \mathcal{L}^* \mathbf{e}_2 \rangle_{X_1}, \\ \langle \mathbf{v}_2, \partial_t \alpha_2 \rangle_{X_2} &= \langle \mathbf{v}_2, \mathbf{L} \mathbf{e}_1 \rangle_{X_2} + \langle \mathbf{v}_2, \mathcal{L} \mathbf{e}_1 \rangle_{X_2}. \end{aligned} \quad (7.16)$$

To obtain a closed system, the constitutive law (7.12b) and the output variables (7.1d) are put in weak form

$$\begin{aligned} \langle \mathbf{v}_1, \mathbf{e}_1 \rangle_{X_1} &= \langle \mathbf{v}_1, \delta_{\alpha_1} H \rangle_{X_1}, \\ \langle \mathbf{v}_2, \mathbf{e}_2 \rangle_{X_2} &= \langle \mathbf{v}_2, \delta_{\alpha_2} H \rangle_{X_2}, \\ \langle \mathbf{v}_\partial, \mathbf{y}_\partial \rangle_{L^2(\partial\Omega)} &= \langle \mathbf{v}_\partial, \mathcal{C}_\partial \mathbf{e} \rangle_{L^2(\partial\Omega)}, \end{aligned} \quad (7.17)$$

where the test function $\mathbf{v}_\partial \in L^2(\partial\Omega, \mathbb{R}^m)$ is defined on the boundary $\partial\Omega$ and \mathcal{C}_∂ is defined either by Eq. (7.10) or (7.11).

Step 2 Next the integration by part has to be carried out. The choice is dictated by the boundary control to be imposed on the system. Consider again Eq. (7.16). The integration by parts can be carried out either on term $-\langle \mathbf{v}_1, \mathcal{L}^* \mathbf{e}_2 \rangle_{X_1}$, or on term $\langle \mathbf{v}_2, \mathcal{L} \mathbf{e}_1 \rangle_{X_2}$. Depending on which line undergoes the integration by parts (this explains the name Partitioned Finite Element method), two structure preserving weak forms are obtained. These differ by the boundary causality imposed to the system.

Integration by parts of the term $-\langle \mathbf{v}_1, \mathcal{L}^* \mathbf{e}_2 \rangle_{X_1}$ In this case case, using Eq. (7.9), it is obtained

$$-\langle \mathbf{v}_1, \mathcal{L}^* \mathbf{e}_2 \rangle_{X_1} = -\langle \mathcal{L} \mathbf{v}_1, \mathbf{e}_2 \rangle_{X_2} + \langle \mathcal{N}_{\partial,1} \mathbf{v}_1, \mathcal{N}_{\partial,2} \mathbf{e}_2 \rangle_{L^2(\partial\Omega)}. \quad (7.18)$$

Then the weak form of the system dynamics reads

$$\begin{aligned} \langle \mathbf{v}_1, \partial_t \boldsymbol{\alpha}_1 \rangle_{X_1} &= -\langle \mathbf{v}_1, \mathbf{L}^\top \mathbf{e}_2 \rangle_{X_1} - \langle \mathcal{L} \mathbf{v}_1, \mathbf{e}_2 \rangle_{X_2} + \langle \mathcal{N}_{\partial,1} \mathbf{v}_1, \mathbf{u}_\partial \rangle_{L^2(\partial\Omega)}, \\ \langle \mathbf{v}, \partial_t \boldsymbol{\alpha} \rangle_X &= \langle \mathbf{v}_2, \mathbf{L} \mathbf{e}_1 \rangle_{X_2} + \langle \mathbf{v}_2, \mathcal{L} \mathbf{e}_1 \rangle_{X_2}, \end{aligned} \quad (7.19)$$

The following proposition is crucial as the lossless character of the infinite-dimensional system (due to the formally skew-adjoint operator) translates into an equivalent property for the corresponding bilinear form in the weak form.

Proposition 8

Given the Hilbert space $H_2^\mathcal{L} := H^\mathcal{L} \times X_2$ and variables $\mathbf{v} = (\mathbf{v}_1, \mathbf{v}_2) \in H_2^\mathcal{L}$, $\mathbf{e} = (\mathbf{e}_1, \mathbf{e}_2) \in H_2^\mathcal{L}$, the bilinear form

$$\begin{aligned} j_\mathcal{L} : H_2^\mathcal{L} \times H_2^\mathcal{L} &\longrightarrow \mathbb{R}, \\ (\mathbf{v}, \mathbf{e}) &\longrightarrow -\langle \mathcal{L} \mathbf{v}_1, \mathbf{e}_2 \rangle_{X_2} + \langle \mathbf{v}_2, \mathcal{L} \mathbf{e}_1 \rangle_{X_2} \end{aligned}$$

is skew-symmetric.

Proof. The proof is obtained by the following computation

$$\begin{aligned} j_\mathcal{L}(\mathbf{v}, \mathbf{e}) &= -\langle \mathcal{L} \mathbf{v}_1, \mathbf{e}_2 \rangle_{X_2} + \langle \mathbf{v}_2, \mathcal{L} \mathbf{e}_1 \rangle_{X_2}, \\ &= -\left(-\langle \mathbf{v}_2, \mathcal{L} \mathbf{e}_1 \rangle_{X_2} + \langle \mathcal{L} \mathbf{v}_1, \mathbf{e}_2 \rangle_{X_2} \right), \\ &= -\left(-\langle \mathcal{L} \mathbf{e}_1, \mathbf{v}_2 \rangle_{X_2} + \langle \mathbf{e}_2, \mathcal{L} \mathbf{v}_1 \rangle_{X_2} \right) = -j_\mathcal{L}(\mathbf{e}, \mathbf{v}). \end{aligned}$$

□

Now assume that the system satisfies the boundary causality condition 7.13. Then, this

choice of the integration by parts lead to the following weak formulation

$$\begin{aligned}
\langle \mathbf{v}_1, \partial_t \boldsymbol{\alpha}_1 \rangle_{X_1} &= -\langle \mathbf{v}_1, \mathbf{L}^\top \mathbf{e}_2 \rangle_{X_1} - \langle \mathcal{L} \mathbf{v}_1, \mathbf{e}_2 \rangle_{X_2} + \langle \mathcal{N}_{\partial,1} \mathbf{v}_1, \mathbf{u}_\partial \rangle_{L^2(\partial\Omega)}, \\
\langle \mathbf{v}_2, \partial_t \boldsymbol{\alpha}_2 \rangle_{X_2} &= \langle \mathbf{v}_2, \mathbf{L} \mathbf{e}_1 \rangle_{X_2} + \langle \mathbf{v}_2, \mathcal{L} \mathbf{e}_1 \rangle_{X_2}, \\
\langle \mathbf{v}_1, \mathbf{e}_1 \rangle_{X_1} &= \langle \mathbf{v}_1, \delta_{\alpha_1} H \rangle_{X_1}, \\
\langle \mathbf{v}_2, \mathbf{e}_2 \rangle_{X_2} &= \langle \mathbf{v}_2, \delta_{\alpha_2} H \rangle_{X_2}, \\
\langle \mathbf{v}_\partial, \mathbf{y}_\partial \rangle_{L^2(\partial\Omega)} &= \langle \mathbf{v}_\partial, \mathcal{N}_{\partial,1} \mathbf{e}_2 \rangle_{L^2(\partial\Omega)}.
\end{aligned} \tag{7.20}$$

Integration by parts of the term $\langle \mathbf{v}_2, \mathcal{L} \mathbf{e}_1 \rangle_{X_2}$ Using Eq. (7.9), it is obtained

$$\langle \mathbf{v}_2, \mathcal{L} \mathbf{e}_1 \rangle_{X_2} = \langle \mathcal{L}^* \mathbf{v}_2, \mathbf{e}_1 \rangle_{X_1} + \langle \mathcal{N}_{\partial,2} \mathbf{v}_2, \mathcal{N}_{\partial,1} \mathbf{e}_1 \rangle_{L^2(\partial\Omega)}. \tag{7.21}$$

Then the weak form of the system dynamics reads

$$\begin{aligned}
\langle \mathbf{v}_1, \partial_t \boldsymbol{\alpha}_1 \rangle_{X_1} &= -\langle \mathbf{v}_1, \mathbf{L}^\top \mathbf{e}_2 \rangle_{X_1} - \langle \mathbf{v}_1, \mathcal{L}^* \mathbf{e}_2 \rangle_{X_1}, \\
\langle \mathbf{v}_2, \partial_t \boldsymbol{\alpha}_2 \rangle_{X_2} &= \langle \mathbf{v}_2, \mathbf{L} \mathbf{e}_1 \rangle_{X_2} + \langle \mathcal{L}^* \mathbf{v}_2, \mathbf{e}_1 \rangle_{X_1} + \langle \mathcal{N}_{\partial,2} \mathbf{v}_2, \mathbf{u}_\partial \rangle_{L^2(\partial\Omega)},
\end{aligned} \tag{7.22}$$

Again the bilinear form arising from the formally skew-adjoint operator is skew-symmetric.

Proposition 9

Given the Hilbert space $H_1^{-\mathcal{L}^*} = X_1 \times H^{-\mathcal{L}^*}$ and variables $\mathbf{v} = (\mathbf{v}_1, \mathbf{v}_2) \in H_1^{-\mathcal{L}^*}$, $\mathbf{e} = (\mathbf{e}_1, \mathbf{e}_2) \in H_1^{-\mathcal{L}^*}$, the bilinear form

$$\begin{aligned}
j_{-\mathcal{L}^*} : H_1^{-\mathcal{L}^*} \times H_1^{-\mathcal{L}^*} &\longrightarrow \mathbb{R}, \\
(\mathbf{v}, \mathbf{e}) &\longrightarrow -\langle \mathbf{v}_1, \mathcal{L}^* \mathbf{e}_2 \rangle_{X_1} + \langle \mathcal{L}^* \mathbf{v}_2, \mathbf{e}_1 \rangle_{X_1}
\end{aligned}$$

is skew-symmetric.

Proof. The proof follows from the computation

$$\begin{aligned}
j_{-\mathcal{L}^*}(\mathbf{v}, \mathbf{e}) &= -\langle \mathbf{v}_1, \mathcal{L}^* \mathbf{e}_2 \rangle_{X_1} + \langle \mathcal{L}^* \mathbf{v}_2, \mathbf{e}_1 \rangle_{X_1}, \\
&= -\left(-\langle \mathcal{L}^* \mathbf{v}_2, \mathbf{e}_1 \rangle_{X_1} + \langle \mathbf{v}_1, \mathcal{L}^* \mathbf{e}_2 \rangle_{X_1} \right), \\
&= -\left(-\langle \mathbf{e}_1, \mathcal{L}^* \mathbf{v}_2 \rangle_{X_1} + \langle \mathcal{L}^* \mathbf{e}_2, \mathbf{v}_1 \rangle_{X_1} \right) = -j_{-\mathcal{L}^*}(\mathbf{e}, \mathbf{v}).
\end{aligned}$$

□

Now assume that the system satisfies the boundary causality condition (7.14). Then, the

1084 final weak formulation reads

$$\begin{aligned}
\langle \mathbf{v}_1, \partial_t \boldsymbol{\alpha}_1 \rangle_{X_1} &= -\langle \mathbf{v}_1, \mathbf{L}^\top \mathbf{e}_2 \rangle_{X_1} - \langle \mathbf{v}_1, \mathcal{L}^* \mathbf{e}_2 \rangle_{X_1}, \\
\langle \mathbf{v}_2, \partial_t \boldsymbol{\alpha}_2 \rangle_{X_2} &= \langle \mathbf{v}_2, \mathbf{L} \mathbf{e}_1 \rangle_{X_2} + \langle \mathcal{L}^* \mathbf{v}_2, \mathbf{e}_1 \rangle_{X_1} + \langle \mathcal{N}_{\partial,2} \mathbf{v}_2, \mathbf{u}_\partial \rangle_{L^2(\partial\Omega)}, \\
\langle \mathbf{v}_1, \mathbf{e}_1 \rangle_{X_1} &= \langle \mathbf{v}_1, \delta_{\alpha_1} H \rangle_{X_1}, \\
\langle \mathbf{v}_2, \mathbf{e}_2 \rangle_{X_2} &= \langle \mathbf{v}_2, \delta_{\alpha_2} H \rangle_{X_2}, \\
\langle \mathbf{v}_\partial, \mathbf{y}_\partial \rangle_{L^2(\partial\Omega)} &= \langle \mathbf{v}_\partial, \mathcal{N}_{\partial,2} \mathbf{e}_2 \rangle_{L^2(\partial\Omega)}.
\end{aligned} \tag{7.23}$$

1085 **Galerkin discretization** To conclude the illustration of this methodology, consider a
 1086 Galerkin discretization is introduced. This means that test, energy and co-energy functions
 1087 are discretized using the same basis. Furthermore the boundary variables are discretized as
 1088 well using bases defined over the boundary

$$\begin{aligned}
\mathbf{v}_1 &\approx \sum_{i=1}^{n_1} \phi_1^i(\mathbf{x}) v_1^i, & \boldsymbol{\alpha}_1 &\approx \sum_{i=1}^{n_1} \phi_1^i(\mathbf{x}) \alpha_1^i(t), & \mathbf{e}_1 &\approx \sum_{i=1}^{n_1} \phi_1^i(\mathbf{x}) e_1^i(t), & \mathbf{x} &\in \Omega, \\
\mathbf{v}_2 &\approx \sum_{i=1}^{n_2} \phi_2^i(\mathbf{x}) v_2^i, & \boldsymbol{\alpha}_2 &\approx \sum_{i=1}^{n_2} \phi_2^i(\mathbf{x}) \alpha_2^i(t), & \mathbf{e}_2 &\approx \sum_{i=1}^{n_2} \phi_2^i(\mathbf{x}) e_2^i(t), & \mathbf{x} &\in \Omega, \\
\mathbf{v}_\partial &\approx \sum_{i=1}^{n_\partial} \phi_\partial^i(s) v_\partial^i, & \mathbf{u}_\partial &\approx \sum_{i=1}^{n_\partial} \phi_\partial^i(s) u_\partial^i(t), & \mathbf{y}_\partial &\approx \sum_{i=1}^{n_\partial} \phi_\partial^i(s) y_\partial^i(t), & s &\in \partial\Omega.
\end{aligned} \tag{7.24}$$

1089 **Discretization of the weak form (7.20)** Plugging the approximation into the weak
 1090 form (7.20) and consider that the resulting equation holds $\forall v_1^i, v_2^j, v_\partial^k$ ($i \in \{1, n_1\}, j \in$
 1091 $\{1, n_2\}, k \in \{1, n_\partial\}$), the finite dimensional system is obtained

$$\begin{aligned}
\begin{bmatrix} \mathbf{M}_1 & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_2 \end{bmatrix} \begin{pmatrix} \dot{\boldsymbol{\alpha}}_{d,1} \\ \dot{\boldsymbol{\alpha}}_{d,2} \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & -\mathbf{D}_0^\top - \mathbf{D}_\mathcal{L}^\top \\ \mathbf{D}_0 + \mathbf{D}_\mathcal{L} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix} + \begin{bmatrix} \mathbf{B}_1 \\ \mathbf{0} \end{bmatrix} \mathbf{u}_\partial, \\
\begin{bmatrix} \mathbf{M}_1 & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_2 \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix} &= \begin{bmatrix} \partial_{\alpha_{d,1}} H_d(\boldsymbol{\alpha}_d) \\ \partial_{\alpha_{d,2}} H_d(\boldsymbol{\alpha}_d) \end{bmatrix}, \\
\mathbf{M}_\partial \mathbf{y}_\partial &= \begin{bmatrix} \mathbf{B}_1^\top & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix}.
\end{aligned} \tag{7.25}$$

1092 Vectors $\boldsymbol{\alpha}_{d,1}, \boldsymbol{\alpha}_{d,2}, \mathbf{e}_1, \mathbf{e}_2, \mathbf{u}_\partial, \mathbf{y}_\partial$ are given by the column-wise concatenation of their respec-
 1093 tive degrees of freedom. The matrices are defined as follows

$$\begin{aligned}
M_1^{ij} &= \langle \phi_1^i, \phi_1^j \rangle_{X_1}, & D_0^{mi} &= \langle \phi_2^m, \mathbf{L} \phi_1^i \rangle_{X_2}, & B_1^{ik} &= \langle \phi_1^i, \phi_\partial^k \rangle_{L^2(\partial\Omega)}, \\
M_2^{mn} &= \langle \phi_2^m, \phi_2^n \rangle_{X_2}, & D_\mathcal{L}^{mi} &= \langle \phi_2^m, \mathcal{L} \phi_1^i \rangle_{X_2}, & M_\partial^{lk} &= \langle \phi_\partial^l, \phi_\partial^k \rangle_{L^2(\partial\Omega)},
\end{aligned} \tag{7.26}$$

where $i, j \in \{1, n_1\}$, $m, n \in \{1, n_2\}$, $l, k \in \{1, n_\partial\}$. Introducing the definitions

$$\begin{aligned}\delta_{\alpha_{d,1}} H_d &:= \delta_{\alpha_1} H \left(\alpha_1 = \sum_{i=1}^{n_1} \phi_1^i \alpha_1^i, \alpha_2 = \sum_{i=1}^{n_1} \phi_2^i \alpha_2^i \right), \\ \delta_{\alpha_{d,2}} H_d &:= \delta_{\alpha_2} H \left(\alpha_1 = \sum_{i=1}^{n_1} \phi_1^i \alpha_1^i, \alpha_2 = \sum_{i=1}^{n_1} \phi_2^i \alpha_2^i \right),\end{aligned}$$

the discretized gradient of the Hamiltonian read

$$\begin{aligned}\partial_{\alpha_{d,1}^i} H_d(\alpha_d) &= \left\langle \phi_1^i, \delta_{\alpha_{d,1}} H_d \right\rangle_{X_1}, \quad i \in \{1, n_1\}, \\ \partial_{\alpha_{d,2}^j} H_d(\alpha_d) &= \left\langle \phi_2^j, \delta_{\alpha_{d,2}} H_d \right\rangle_{X_2}, \quad j \in \{1, n_2\}.\end{aligned}\tag{7.27}$$

A pH system in canonical form is found observing that Sys. (7.25) is compactly rewritten as

$$\mathbf{M} \dot{\alpha}_d = \mathbf{J}_{\mathcal{L}} \mathbf{e} + \mathbf{B} \mathbf{u}_\partial, \tag{7.28}$$

$$\mathbf{M} \mathbf{e} = \nabla H_d(\alpha_d), \tag{7.29}$$

$$\mathbf{M}_\partial \mathbf{y}_\partial = \mathbf{B}^\top \mathbf{e}, \tag{7.30}$$

where $\alpha_d = (\alpha_{d,1}^\top \ \alpha_{d,2}^\top)^\top$, $\mathbf{e} = (\mathbf{e}_1^\top \ \mathbf{e}_2^\top)^\top$, $\nabla H_d(\alpha_d) = (\partial_{\alpha_{d,1}}^\top H_d(\alpha_d) \ \partial_{\alpha_{d,2}}^\top H_d(\alpha_d))^\top$ and

$$\mathbf{M} = \begin{bmatrix} \mathbf{M}_1 & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_2 \end{bmatrix}, \quad \mathbf{J}_{\mathcal{L}} = \begin{bmatrix} \mathbf{0} & -\mathbf{D}_0^\top - \mathbf{D}_{\mathcal{L}}^\top \\ \mathbf{D}_0 + \mathbf{D}_{\mathcal{L}} & \mathbf{0} \end{bmatrix}, \quad \mathbf{B} = \begin{bmatrix} \mathbf{B}_1 \\ \mathbf{0} \end{bmatrix}. \tag{7.31}$$

Plugging (7.29) into (7.28), a pH system in canonical form is obtained

$$\begin{aligned}\dot{\alpha}_d &= \mathbf{J} \nabla H_d(\alpha_d) + \mathbf{B} \mathbf{u}_\partial, \quad \text{where} \quad \mathbf{J} = \mathbf{M}^{-1} \mathbf{J}_{\mathcal{L}} \mathbf{M}^{-1}, \\ \hat{\mathbf{y}}_\partial &= \mathbf{B}^\top \nabla H_d(\alpha_d), \quad \text{where} \quad \hat{\mathbf{y}}_\partial = \mathbf{M}_\partial \mathbf{y}_\partial.\end{aligned}\tag{7.32}$$

The structure preserving character of the method is evident from the preservation at the discrete level of the power balance. The finite dimensional counterpart of the energy rate is given by

$$\begin{aligned}\dot{H}_d &= \nabla^\top H_d(\alpha_d) \dot{\alpha}_d, \\ &= \nabla^\top H_d(\alpha_d) \mathbf{J} \nabla H_d(\alpha_d) + \nabla^\top H_d(\alpha_d) \mathbf{B} \mathbf{u}_\partial, \quad \text{Skew-symmetry of } \mathbf{J} \\ &= \hat{\mathbf{y}}_\partial^\top \mathbf{u}_\partial.\end{aligned}\tag{7.33}$$

This result mimics its infinite dimensional equivalent (7.15).

Discretization of the weak form (7.23) Plugging the approximation into the weak form (7.23) a finite dimensional system with a different causality is obtained

$$\begin{aligned}
\begin{bmatrix} \mathbf{M}_1 & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_2 \end{bmatrix} \begin{pmatrix} \dot{\boldsymbol{\alpha}}_{d,1} \\ \dot{\boldsymbol{\alpha}}_{d,2} \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & -\mathbf{D}_0^\top + \mathbf{D}_{-\mathcal{L}^*} \\ \mathbf{D}_0 - \mathbf{D}_{-\mathcal{L}^*}^\top & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix} + \begin{bmatrix} \mathbf{0} \\ \mathbf{B}_2 \end{bmatrix} \mathbf{u}_\partial, \\
\begin{bmatrix} \mathbf{M}_1 & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_2 \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix} &= \begin{pmatrix} \partial_{\boldsymbol{\alpha}_{d,1}} H_d(\boldsymbol{\alpha}_d) \\ \partial_{\boldsymbol{\alpha}_{d,2}} H_d(\boldsymbol{\alpha}_d) \end{pmatrix}, \\
\mathbf{M}_\partial \mathbf{y}_\partial &= \begin{bmatrix} \mathbf{0} & \mathbf{B}_2^\top \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix}.
\end{aligned} \tag{7.34}$$

The differences with respect to formulation (7.25) reside in matrices $\mathbf{D}_{-\mathcal{L}^*}$, \mathbf{B}_2 , whose definitions are

$$D_{-\mathcal{L}^*}^{im} = \langle \phi_1^i, -\mathcal{L}^* \phi_2^m \rangle_{X_1}, \quad B_2^{mk} = \langle \phi_2^m, \phi_\partial^k \rangle_{X_{\partial\Omega}} \quad i \in \{1, n_1\}, m \in \{1, n_2\}, k \in \{1, n_\partial\}. \tag{7.35}$$

System (7.34) can be put in canonical form by replacing the co-energy variables by the discretized gradient.

Example: the irrotational shallow water equations Consider as an example the shallow water equations detailed in Sec. §3.2.3. The flow is assumed to be irrotational ($\nabla \times \mathbf{v} = 0$). As a consequence the term \mathcal{G} in Eq. (3.29) vanishes. To fulfill Assumption 2, the incoming volumetric flow is known at the boundary, so that a uniform Neumann condition is imposed. This lead to the following boundary control system, defined on an open connected set $\Omega \subset \mathbb{R}^2$

$$\begin{aligned}
\frac{\partial}{\partial t} \begin{pmatrix} \alpha_h \\ \boldsymbol{\alpha}_v \end{pmatrix} &= - \begin{bmatrix} 0 & \text{div} \\ \text{grad} & \mathbf{0} \end{bmatrix} \begin{pmatrix} e_h \\ \mathbf{e}_v \end{pmatrix}, & \alpha_h &\in L^2(\Omega), \\
& & \boldsymbol{\alpha}_v &\in L^2(\Omega, \mathbb{R}^2), \\
\begin{pmatrix} e_h \\ \mathbf{e}_v \end{pmatrix} &:= \begin{pmatrix} \delta_{\alpha_h} H \\ \delta_{\boldsymbol{\alpha}_v} H \end{pmatrix} = \begin{pmatrix} \frac{1}{2\rho} \|\boldsymbol{\alpha}_v\|^2 + \rho g \alpha_h \\ \frac{1}{\rho} \boldsymbol{\alpha}_h \boldsymbol{\alpha}_v \end{pmatrix}, & e_h &\in H^1(\Omega), \\
& & \mathbf{e}_v &\in H^{\text{div}}(\Omega, \mathbb{R}^2), \\
u_\partial &= -\mathbf{e}_v \cdot \mathbf{n}, & u_\partial &\in \mathbb{R}, \\
y_\partial &= e_h, & y_\partial &\in \mathbb{R},
\end{aligned} \tag{7.36}$$

where the Hamiltonian is a non linear functional in the energy variables

$$H(\alpha_h, \boldsymbol{\alpha}_v) = \frac{1}{2} \int_\Omega \left\{ \frac{1}{\rho} \alpha_h \|\boldsymbol{\alpha}_v\|^2 + \rho g \alpha_h^2 \right\} d\Omega.$$

The energy and co-energy variables are related to the physical variables (fluid height and velocity) through Eqs. (3.26), (3.28). In this case $X_1 = L^2(\Omega)$, $X_2 = L^2(\Omega, \mathbb{R}^2)$ and $\mathcal{L} = \text{grad}$, $-\mathcal{L}^* = \text{div}$. This implies $H^\mathcal{L} = H^1(\Omega)$, $H^{-\mathcal{L}^*} = H^{\text{div}}(\Omega, \mathbb{R}^2)$. As shown in (3.30), the energy rate is given by

$$\dot{H} = -\langle \mathbf{e}_v, \text{grad } e_h \rangle_{L^2(\Omega, \mathbb{R}^2)} - \langle \text{div } \mathbf{e}_v, e_h \rangle_{L^2(\Omega)} = \langle -\mathbf{e}_v \cdot \mathbf{n}, e_h \rangle_{L^2(\partial\Omega)}. \tag{7.37}$$

1116 The boundary operators are therefore given by

$$\begin{aligned} u_\partial &= \mathcal{N}_{\partial,2} \mathbf{e}_v = -\gamma_n \mathbf{e}_v = -\mathbf{e}_v \cdot \mathbf{n}|_{\partial\Omega}, \\ y_\partial &= \mathcal{N}_{\partial,1} e_h = \gamma_0 e_h = e_h|_{\partial\Omega}. \end{aligned} \quad (7.38)$$

1117 This system represents a particular example of the general formulation of the general frame-
1118 work (7.12), together with boundary conditions (7.13). To obtain a finite dimensional system,
1119 the test variables v_h , \mathbf{v}_v are introduced and the integration by parts is performed on the div
1120 operator, leading to the weak form

$$\begin{aligned} \langle v_h, \partial_t \alpha_h \rangle_{L^2(\Omega)} &= \langle \text{grad } v_h, \mathbf{e}_v \rangle_{L^2(\Omega, \mathbb{R}^2)} + \langle \gamma_0 v_h, \mathbf{u}_\partial \rangle_{L^2(\partial\Omega)}, \\ \langle \mathbf{v}_v, \partial_t \alpha_v \rangle_{L^2(\Omega, \mathbb{R}^2)} &= -\langle \mathbf{v}_v, \text{grad } e_h \rangle_{L^2(\Omega, \mathbb{R}^2)}, \\ \langle v_h, e_h \rangle_{L^2(\Omega)} &= \left\langle v_h, \frac{1}{2\rho} \|\alpha_v\|^2 + \rho g \alpha_h \right\rangle_{L^2(\Omega)}, \\ \langle \mathbf{v}_v, \mathbf{e}_v \rangle_{L^2(\Omega, \mathbb{R}^2)} &= \left\langle \mathbf{v}_v, \frac{1}{\rho} \alpha_h \alpha_v \right\rangle_{L^2(\Omega, \mathbb{R}^2)}, \\ \langle v_\partial, y_\partial \rangle_{L^2(\partial\Omega)} &= \langle v_\partial, \gamma_0 e_h \rangle_{L^2(\partial\Omega)}. \end{aligned} \quad (7.39)$$

1121 Introducing a Galerkin approximation as in (7.24)

$$\begin{aligned} v_h &\approx \sum_{i=1}^{n_h} \phi_h^i(\mathbf{x}) v_h^i, & \alpha_h &\approx \sum_{i=1}^{n_h} \phi_h^i(\mathbf{x}) \alpha_h^i(t), & e_h &\approx \sum_{i=1}^{n_h} \phi_h^i(\mathbf{x}) e_h^i(t), & \mathbf{x} &\in \Omega, \\ \mathbf{v}_v &\approx \sum_{i=1}^{n_v} \phi_v^i(\mathbf{x}) \mathbf{v}_v^i, & \alpha_v &\approx \sum_{i=1}^{n_v} \phi_v^i(\mathbf{x}) \alpha_v^i(t), & \mathbf{e}_v &\approx \sum_{i=1}^{n_v} \phi_v^i(\mathbf{x}) \mathbf{e}_v^i(t), & \mathbf{x} &\in \Omega, \\ v_\partial &\approx \sum_{i=1}^{n_\partial} \phi_\partial^i(s) v_\partial^i, & u_\partial &\approx \sum_{i=1}^{n_\partial} \phi_\partial^i(s) u_\partial^i(t), & y_\partial &\approx \sum_{i=1}^{n_\partial} \phi_\partial^i(s) y_\partial^i(t), & s &\in \partial\Omega, \end{aligned} \quad (7.40)$$

1122 the finite dimensional system is obtained

$$\begin{aligned} \begin{bmatrix} \mathbf{M}_h & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_v \end{bmatrix} \begin{pmatrix} \dot{\alpha}_{d,h} \\ \dot{\alpha}_{d,v} \end{pmatrix} &= - \begin{bmatrix} \mathbf{0} & -\mathbf{D}_{\text{grad}}^\top \\ \mathbf{D}_{\text{grad}} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_h \\ \mathbf{e}_v \end{pmatrix} + \begin{bmatrix} \mathbf{B}_h \\ \mathbf{0} \end{bmatrix} \mathbf{u}_\partial, \\ \begin{bmatrix} \mathbf{M}_h & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_v \end{bmatrix} \begin{pmatrix} \mathbf{e}_h \\ \mathbf{e}_v \end{pmatrix} &= \begin{bmatrix} \partial_{\alpha_{d,h}} H_d(\alpha_{d,h}, \alpha_{d,v}) \\ \partial_{\alpha_{d,v}} H_d(\alpha_{d,h}, \alpha_{d,v}) \end{bmatrix}, \\ \mathbf{M}_{\partial} \mathbf{y}_\partial &= \begin{bmatrix} \mathbf{B}_h^\top & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_h \\ \mathbf{e}_v \end{pmatrix}. \end{aligned} \quad (7.41)$$

1123 The matrices are defined as follows

$$\begin{aligned} M_h^{ij} &= \langle \phi_h^i, \phi_h^j \rangle_{L^2(\Omega)}, & D_{\text{grad}}^{mi} &= \langle \phi_v^m, \text{grad } \phi_h^i \rangle_{L^2(\Omega, \mathbb{R}^2)}, \\ M_v^{mn} &= \langle \phi_v^m, \phi_v^n \rangle_{L^2(\Omega, \mathbb{R}^2)}, & B_h^{ik} &= \langle \phi_h^i, \phi_\partial^k \rangle_{L^2(\partial\Omega)}, \\ M_\partial^{lk} &= \langle \phi_\partial^l, \phi_\partial^k \rangle_{L^2(\partial\Omega)}, \end{aligned} \quad (7.42)$$

where $i, j \in \{1, n_h\}$, $m, n \in \{1, n_v\}$, $l, k \in \{1, n_\partial\}$. The discretized gradient of the Hamiltonian read

$$\begin{aligned} \partial_{\alpha_{d,h}^i} H_d(\alpha_{d,h}, \alpha_{d,v}) &= \left\langle \phi_h^i, \frac{1}{2\rho} \left\| \sum_{r=1}^{n_2} \phi_v^r \alpha_v^r \right\|^2 + \rho g \sum_{r=1}^{n_1} \phi_h^r \alpha_h^r \right\rangle_{L^2(\Omega)}, \quad i \in \{1, n_h\}, \\ \partial_{\alpha_{d,v}^m} H_d(\alpha_{d,h}, \alpha_{d,v}) &= \left\langle \phi_v^m, \frac{1}{\rho} \left(\sum_{r=1}^{n_1} \phi_h^r \alpha_h^r \right) \left(\sum_{r=1}^{n_2} \phi_v^r \alpha_v^r \right) \right\rangle_{L^2(\Omega, \mathbb{R}^2)}, \quad m \in \{1, n_v\}. \end{aligned} \quad (7.43)$$

One possible finite element discretization for this problem can be found in [Pir89]. The non linear nature of the problem strongly complicates the analysis. The presence of shocks has to be accounted for in the numerical discretization. The proposed methodology has to cope with finite time shocks to become a valid alternative to already well established strategies.

7.1.2 Linear case

The general framework detailed in Sec. 7.1.1 is valid for both linear and non linear system. However, in the linear case a major simplification occurs since the constitutive law connecting energy and co-energy variables is easily invertible. This allows using only to describe the dynamics using the co-energy variables only.

The additional assumption required to make the system linear is introduced.

Assumption 3

The Hamiltonian is assumed to be a positive quadratic functional in the energy variables α_1, α_2 . Furthermore, the Hamiltonian is considered to be separable with respect to α_1, α_2 (this hypothesis is always met for the systems under consideration). Therefore, it can be expressed as

$$H = \frac{1}{2} \langle \alpha_1, \mathcal{Q}_1 \alpha_1 \rangle_{X_1} + \frac{1}{2} \langle \alpha_2, \mathcal{Q}_2 \alpha_2 \rangle_{X_2}, \quad (7.44)$$

where $\mathcal{Q}_1, \mathcal{Q}_2$ are symmetric operators, bounded from below and above

$$m_1 \mathbf{I}_1 \leq \mathcal{Q}_1 \leq M_1 \mathbf{I}_1, \quad m_2 \mathbf{I}_2 \leq \mathcal{Q}_2 \leq M_2 \mathbf{I}_2.$$

Because of assumption 3, the co-energy variables are given by

$$e_1 := \delta_{\alpha_1} H = \mathcal{Q}_1 \alpha_1, \quad e_2 := \delta_{\alpha_2} H = \mathcal{Q}_2 \alpha_2 \quad (7.45)$$

Since $\mathcal{Q}_1, \mathcal{Q}_2$ are bounded from below and above operators, it is possible to invert them to obtain

$$\alpha_1 = \mathcal{Q}_1^{-1} e_1 = \mathcal{M}_1 e_1, \quad \alpha_2 = \mathcal{Q}_2^{-1} e_2 = \mathcal{M}_2 e_2, \quad \mathcal{M}_1 := \mathcal{Q}_1^{-1}, \mathcal{M}_2 := \mathcal{Q}_2^{-1}. \quad (7.46)$$

1145 The Hamiltonian is then written in terms of co-energy variables as

$$H = \frac{1}{2} \langle \mathbf{e}_1, \mathcal{M}_1 \mathbf{e}_1 \rangle_{X_1} + \frac{1}{2} \langle \mathbf{e}_2, \mathcal{M}_2 \mathbf{e}_2 \rangle_{X_2}. \quad (7.47)$$

1146 Under assumptions 1, 2, 3, a pH linear system is expressed as

$$\begin{bmatrix} \mathcal{M}_1 & 0 \\ 0 & \mathcal{M}_2 \end{bmatrix} \partial_t \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix} = \begin{bmatrix} 0 & -\mathbf{L}^\top - \mathcal{L}^* \\ \mathbf{L} + \mathcal{L} & 0 \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix}, \quad \begin{matrix} \mathbf{e}_1 \in H^\mathcal{L}, \\ \mathbf{e}_2 \in H^{-\mathcal{L}^*}. \end{matrix} \quad (7.48)$$

1147 If Eq. (7.10) holds the boundary variables equal

$$\mathbf{u}_\partial = \mathcal{N}_2 \mathbf{e}_2, \quad \mathbf{y}_\partial = \mathcal{N}_1 \mathbf{e}_1, \quad \mathbf{u}_\partial, \mathbf{y}_\partial \in \mathbb{R}^m. \quad (7.49)$$

1148 Whereas if Eq. (7.11) holds, then

$$\mathbf{u}_\partial = \mathcal{N}_1 \mathbf{e}_1, \quad \mathbf{y}_\partial = \mathcal{N}_2 \mathbf{e}_2, \quad \mathbf{u}_\partial, \mathbf{y}_\partial \in \mathbb{R}^m. \quad (7.50)$$

1149 From equation (7.47), the power balance reads

$$\begin{aligned} \dot{H} &= \langle \mathbf{e}_1, \mathcal{M}_1 \partial_t \mathbf{e}_1 \rangle_{X_1} + \langle \mathbf{e}_2, \mathcal{M}_2 \partial_t \mathbf{e}_2 \rangle_{X_2}, \\ &= \langle \mathbf{e}_1, -\mathcal{L}^* \mathbf{e}_2 \rangle_{X_1} + \langle \mathbf{e}_2, \mathcal{L} \mathbf{e}_1 \rangle_{X_2}, \\ &= \langle \mathcal{N}_{\partial,1} \mathbf{e}_1, \mathcal{N}_{\partial,2} \mathbf{e}_2 \rangle_{L^2(\partial\Omega)}, \\ &= \langle \mathbf{y}_\partial, \mathbf{u}_\partial \rangle_{L^2(\partial\Omega)}. \end{aligned} \quad (7.51)$$

1150 To get a finite dimensional approximation the same procedure detailed in Sec. §7.1.1 is
1151 followed. The only difference is that there is no need to discretize the constitutive relations
1152 as those are already incorporated in the dynamics.

1153 Once the system is put into weak form, if the operator $-\mathcal{L}^*$ is integrated by parts, one
1154 obtains the weak form

$$\begin{aligned} \langle \mathbf{v}_1, \mathcal{M}_1 \partial_t \mathbf{e}_1 \rangle_{X_1} &= -\langle \mathbf{v}_1, \mathbf{L}^\top \mathbf{e}_2 \rangle_{X_1} - \langle \mathcal{L} \mathbf{v}_1, \mathbf{e}_2 \rangle_{X_2} + \langle \mathcal{N}_{\partial,1} \mathbf{v}_1, \mathbf{u}_\partial \rangle_{L^2(\partial\Omega)}, \\ \langle \mathbf{v}_2, \mathcal{M}_2 \partial_t \mathbf{e}_2 \rangle_{X_2} &= \langle \mathbf{v}_2, \mathbf{L} \mathbf{e}_1 \rangle_{X_2} + \langle \mathbf{v}_2, \mathcal{L} \mathbf{e}_1 \rangle_{X_2}, \\ \langle \mathbf{v}_\partial, \mathbf{y}_\partial \rangle_{L^2(\partial\Omega)} &= \langle \mathbf{v}_\partial, \mathcal{N}_{\partial,1} \mathbf{e}_2 \rangle_{L^2(\partial\Omega)}. \end{aligned} \quad (7.52)$$

1155 Otherwise, if operator \mathcal{L} is integrated by parts, it is found

$$\begin{aligned} \langle \mathbf{v}_1, \mathcal{M}_1 \partial_t \mathbf{e}_1 \rangle_{X_1} &= -\langle \mathbf{v}_1, \mathbf{L}^\top \mathbf{e}_2 \rangle_{X_1} - \langle \mathbf{v}_1, \mathcal{L}^* \mathbf{e}_2 \rangle_{X_1}, \\ \langle \mathbf{v}_2, \mathcal{M}_2 \partial_t \mathbf{e}_2 \rangle_{X_2} &= \langle \mathbf{v}_2, \mathbf{L} \mathbf{e}_1 \rangle_{X_2} + \langle \mathcal{L}^* \mathbf{v}_2, \mathbf{e}_1 \rangle_{X_1} + \langle \mathcal{N}_{\partial,2} \mathbf{v}_2, \mathbf{u}_\partial \rangle_{L^2(\partial\Omega)}, \\ \langle \mathbf{v}_\partial, \mathbf{y}_\partial \rangle_{L^2(\partial\Omega)} &= \langle \mathbf{v}_\partial, \mathcal{N}_{\partial,2} \mathbf{e}_2 \rangle_{L^2(\partial\Omega)}. \end{aligned} \quad (7.53)$$

1156 After introducing a Galerkin approximation as in (7.24), the discretized version of the weak
1157 form (7.52) reads

$$\begin{aligned} \begin{bmatrix} \mathbf{M}_{\mathcal{M}_1} & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_{\mathcal{M}_2} \end{bmatrix} \begin{pmatrix} \dot{\mathbf{e}}_1 \\ \dot{\mathbf{e}}_2 \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & -\mathbf{D}_0^\top - \mathbf{D}_{\mathcal{L}}^\top \\ \mathbf{D}_0 + \mathbf{D}_{\mathcal{L}} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix} + \begin{bmatrix} \mathbf{B}_1 \\ \mathbf{0} \end{bmatrix} \mathbf{u}_\partial, \\ \mathbf{M}_\partial \mathbf{y}_\partial &= \begin{bmatrix} \mathbf{B}_1^\top & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix}. \end{aligned} \quad (7.54)$$

1158 The only difference with respect to Eq. (7.25) concerns the mass matrices

$$M_{\mathcal{M}_1}^{ij} = \langle \phi_1^i, \mathcal{M}_1 \phi_1^j \rangle_{X_1}, \quad M_{\mathcal{M}_2}^{mn} = \langle \phi_2^m, \mathcal{M}_2 \phi_2^n \rangle_{X_2} \quad i, j \in \{1, n_1\}, \quad m, n \in \{1, n_2\}. \quad (7.55)$$

1159 If the Galerkin approximation is applied to the weak form (7.53), it is obtained

$$\begin{aligned} \begin{bmatrix} \mathbf{M}_{\mathcal{M}_1} & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_{\mathcal{M}_2} \end{bmatrix} \begin{pmatrix} \dot{\mathbf{e}}_1 \\ \dot{\mathbf{e}}_2 \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & -\mathbf{D}_0^\top + \mathbf{D}_{-\mathcal{L}^*} \\ \mathbf{D}_0 - \mathbf{D}_{-\mathcal{L}^*}^\top & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix} + \begin{bmatrix} \mathbf{0} \\ \mathbf{B}_2 \end{bmatrix} \mathbf{u}_\partial, \\ \mathbf{M}_\partial \mathbf{y}_\partial &= \begin{bmatrix} \mathbf{0} & \mathbf{B}_2^\top \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix}. \end{aligned} \quad (7.56)$$

1160 In both cases, it is easy to verify that the Hamiltonian

$$H_d = \frac{1}{2} \mathbf{e}_1^\top \mathbf{M}_{\mathcal{M}_1} \mathbf{e}_1 + \frac{1}{2} \mathbf{e}_2^\top \mathbf{M}_{\mathcal{M}_2} \mathbf{e}_2, \quad (7.57)$$

1161 once differentiated in time, provides the energy rate

$$\dot{H}_d = \mathbf{y}_\partial^\top \mathbf{M}_\partial \mathbf{u}_\partial = \hat{\mathbf{y}}_\partial^\top \mathbf{u}_\partial, \quad \text{where} \quad \hat{\mathbf{y}}_\partial := \mathbf{M}_\partial \mathbf{y}_\partial. \quad (7.58)$$

1162 This result mimics its finite dimensional counterpart (7.51).

1163 7.1.3 Linear flexible structures

1164 In this section, some linear example from the elasticity realms are considered. We restrict
1165 the discussion to linear problems. This case is anyway significant, as these examples are
1166 frequently encountered in engineering applications.

1167 7.1.3.1 Euler-Bernoulli beam

1168 We reconsider the example discussed in Sec. §3.2.1. The relation between energy and co-
1169 energy variables is given by Eqs. (3.20), (3.22)

$$\alpha_w = \rho A e_w, \quad \alpha_\kappa = \frac{1}{EI} e_\kappa \quad (7.59)$$

1170 The coefficients ρ, A, E and I are the mass density, the cross section area, Young's modulus
1171 of elasticity and the moment of inertia of the cross section.

1172 **Control through forces and torques** Given an interval $\Omega = (0, L)$, a thin beam under
 1173 free boundary condition (forces and torques imposed at the boundary) can be modeled in
 1174 terms of co-energy variables by the following system

$$\begin{bmatrix} \rho A & 0 \\ 0 & (EI)^{-1} \end{bmatrix} \frac{\partial}{\partial t} \begin{pmatrix} e_w \\ e_\kappa \end{pmatrix} = \begin{bmatrix} 0 & -\partial_{xx} \\ \partial_{xx} & 0 \end{bmatrix} \begin{pmatrix} e_w \\ e_\kappa \end{pmatrix}, \quad \begin{matrix} e_w \in H^2(\Omega), \\ e_\kappa \in H^2(\Omega), \end{matrix} \quad (7.60a)$$

$$\mathbf{u}_\partial = \begin{bmatrix} 0 & \gamma_0 \\ 0 & -\gamma_1 \end{bmatrix} \begin{pmatrix} e_w \\ e_\kappa \end{pmatrix}, \quad \mathbf{u}_\partial \in \mathbb{R}^4, \quad (7.60b)$$

$$\mathbf{y}_\partial = \begin{bmatrix} \gamma_1 & 0 \\ \gamma_0 & 0 \end{bmatrix} \begin{pmatrix} e_w \\ e_\kappa \end{pmatrix}, \quad \mathbf{y}_\partial \in \mathbb{R}^4. \quad (7.60c)$$

1175 The boundary operator γ_0, γ_1 denote the trace and the first derivative trace along the bound-
 1176 ary. In a one-dimensional domain the boundary degenerates to two single points

$$\gamma_0 a = a|_{\partial\Omega} = \begin{pmatrix} -a(0) \\ +a(L) \end{pmatrix}, \quad \gamma_1 a = \partial_x a|_{\partial\Omega} = \begin{pmatrix} -\partial_x a(0) \\ +\partial_x a(L) \end{pmatrix}. \quad (7.61)$$

1177 In this case $X_1 = X_2 = L^2(\Omega)$. The operators $\mathcal{L}, N_{\partial,1}, N_{\partial,2}$ read

$$\mathcal{L} = \partial_{xx}, \quad N_{\partial,1} = \begin{bmatrix} \gamma_1 \\ \gamma_0 \end{bmatrix}, \quad N_{\partial,2} = \begin{bmatrix} \gamma_0 \\ -\gamma_1 \end{bmatrix}. \quad (7.62)$$

1178 The Hamiltonian is given by

$$H = \frac{1}{2} \int_{\Omega} \left\{ \rho A e_w^2 + (EI)^{-1} e_\kappa^2 \right\} d\Omega. \quad (7.63)$$

1179 Applying twice the integration by parts formula, one obtains the power balance

$$\begin{aligned} \dot{H} &= \langle e_w, \rho A \partial_t e_w \rangle_{L^2(\Omega)} + \langle e_\kappa, (EI)^{-1} \partial_t e_\kappa \rangle_{L^2(\Omega)}, \\ &= \langle e_w, -\partial_{xx} e_\kappa \rangle_{L^2(\Omega)} + \langle e_\kappa, \partial_{xx} e_w \rangle_{L^2(\Omega)}, \\ &= \langle \gamma_1 e_w, \gamma_0 e_\kappa \rangle_{\mathbb{R}^2} + \langle \gamma_0 e_w, -\gamma_1 e_\kappa \rangle_{\mathbb{R}^2}, \\ &= \langle \mathbf{y}_\partial, \mathbf{u}_\partial \rangle_{\mathbb{R}^4}. \end{aligned} \quad (7.64)$$

1180 Given the test functions v_w, v_κ , the weak form is readily obtained as

$$\begin{aligned} \langle v_w, \rho A \partial_t e_w \rangle_{L^2(\Omega)} &= \langle v_w, -\partial_{xx} e_\kappa \rangle_{L^2(\Omega)}, \\ \langle v_\kappa, (EI)^{-1} \partial_t e_\kappa \rangle_{L^2(\Omega)} &= \langle v_\kappa, \partial_{xx} e_w \rangle_{L^2(\Omega)}. \end{aligned} \quad (7.65)$$

1181 If the integration by parts is applied twice to the first line of Eq. (7.60a), it is obtained

$$\begin{aligned} \langle v_w, \rho A \partial_t e_w \rangle_{L^2(\Omega)} &= -\langle \partial_{xx} v_w, e_\kappa \rangle_{L^2(\Omega)} + \langle \gamma_1 v_w, (u_{\partial,1}, u_{\partial,2}) \rangle_{\mathbb{R}^2} + \langle \gamma_0 v_w, (u_{\partial,3}, u_{\partial,4}) \rangle_{\mathbb{R}^2}, \\ \langle v_\kappa, (EI)^{-1} \partial_t e_\kappa \rangle_{L^2(\Omega)} &= \langle v_\kappa, \partial_{xx} e_w \rangle_{L^2(\Omega)}. \end{aligned} \quad (7.66)$$

1182 Introducing a Galerkin discretization for test and efforts functions

$$v_w = \sum_{i=1}^{n_w} \phi_w^i v_w^i, \quad e_w = \sum_{i=1}^{n_w} \phi_w^i e_w^i(t), \quad v_\kappa = \sum_{i=1}^{n_\kappa} \phi_\kappa^i v_\kappa^i, \quad e_\kappa = \sum_{i=1}^{n_\kappa} \phi_\kappa^i e_\kappa^i(t), \quad (7.67)$$

1183 and considering that $\mathbf{u}_\partial \in \mathbb{R}^4$, $\mathbf{y}_\partial \in \mathbb{R}^4$, the following is obtained

$$\begin{aligned} \begin{bmatrix} \mathbf{M}_{\rho A} & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_{EI^{-1}} \end{bmatrix} \begin{pmatrix} \dot{\mathbf{e}}_w \\ \dot{\mathbf{e}}_\kappa \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & -\mathbf{D}_{\partial xx}^\top \\ \mathbf{D}_{\partial xx} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_w \\ \mathbf{e}_\kappa \end{pmatrix} + \begin{bmatrix} \mathbf{B}_w \\ \mathbf{0} \end{bmatrix} \mathbf{u}_\partial, \\ \mathbf{y}_\partial &= \begin{bmatrix} \mathbf{B}_w^\top & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_w \\ \mathbf{e}_\kappa \end{pmatrix}. \end{aligned} \quad (7.68)$$

1184 The matrices $\mathbf{M}_{\rho A}$, $\mathbf{M}_{EI^{-1}}$, $\mathbf{D}_{\partial xx}$ are defined as ($i, j \in \{1, n_w\}$, $m, n \in \{1, n_\kappa\}$)

$$M_{\rho A}^{ij} = \langle \phi_w^i, \rho A \phi_w^j \rangle_{L^2(\Omega)}, \quad M_{EI^{-1}}^{mn} = \langle \phi_\kappa^m, (EI)^{-1} \phi_\kappa^n \rangle_{L^2(\Omega)}, \quad D_{\partial xx}^{mi} = \langle \phi_\kappa^m, \partial_{xx} \phi_w^i \rangle_{L^2(\Omega)}. \quad (7.69)$$

1185 The \mathbf{B}_w is composed of four column vectors $\mathbf{B}_w = [\mathbf{b}_w^1 \ \mathbf{b}_w^2 \ \mathbf{b}_w^3 \ \mathbf{b}_w^4]$

$$b_w^{1,i} = -\partial_x \phi_w^i(0), \quad b_w^{2,i} = \partial_x \phi_w^i(L), \quad b_w^{3,i} = -\phi_w^i(0), \quad b_w^{4,i} = \phi_w^i(L), \quad i \in \{1, n_w\}. \quad (7.70)$$

Control through linear and angular velocities Equivalently, the second line of Eq. (7.60a) could have been integrated by parts to control through the linear and angular velocities at the extremities. Consider the system with known forces and torques at the extremities

$$\begin{bmatrix} \rho A & 0 \\ 0 & (EI)^{-1} \end{bmatrix} \frac{\partial}{\partial t} \begin{pmatrix} e_w \\ e_\kappa \end{pmatrix} = \begin{bmatrix} 0 & -\partial_{xx} \\ \partial_{xx} & 0 \end{bmatrix} \begin{pmatrix} e_w \\ e_\kappa \end{pmatrix}, \quad \begin{aligned} e_w &\in H^2(\Omega), \\ e_\kappa &\in H^2(\Omega), \end{aligned} \quad (7.71a)$$

$$\mathbf{u}_\partial = \begin{bmatrix} \gamma_1 & 0 \\ \gamma_0 & 0 \end{bmatrix} \begin{pmatrix} e_w \\ e_\kappa \end{pmatrix}, \quad \mathbf{u}_\partial \in \mathbb{R}^4, \quad (7.71b)$$

$$\mathbf{y}_\partial = \begin{bmatrix} 0 & \gamma_0 \\ 0 & -\gamma_1 \end{bmatrix} \begin{pmatrix} e_w \\ e_\kappa \end{pmatrix}, \quad \mathbf{y}_\partial \in \mathbb{R}^4. \quad (7.71c)$$

1186 Once the system is put into weak form and the second line of Eq. (7.71a) is integrated twice,
1187 it is computed

$$\begin{aligned} \langle v_w, \rho A \partial_t e_w \rangle_{L^2(\Omega)} &= \langle v_w, -\partial_{xx} e_\kappa \rangle_{L^2(\Omega)}, \\ \langle v_\kappa, (EI)^{-1} \partial_t e_\kappa \rangle_{L^2(\Omega)} &= \langle \partial_{xx} v_\kappa, e_w \rangle_{L^2(\Omega)} + \langle \gamma_0 v_\kappa, (u_{\partial,1}, u_{\partial,2}) \rangle_{\mathbb{R}^2} + \langle -\gamma_1 v_\kappa, (u_{\partial,3}, u_{\partial,4}) \rangle_{\mathbb{R}^2}. \end{aligned} \quad (7.72)$$

1188 Replacing a Galerkin approximation, it is obtained

$$\begin{aligned} \begin{bmatrix} \mathbf{M}_{\rho A} & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_{EI^{-1}} \end{bmatrix} \begin{pmatrix} \dot{\mathbf{e}}_w \\ \dot{\mathbf{e}}_\kappa \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & \mathbf{D}_{-\partial_{xx}} \\ -\mathbf{D}_{-\partial_{xx}}^\top & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_w \\ \mathbf{e}_\kappa \end{pmatrix} + \begin{bmatrix} \mathbf{0} \\ \mathbf{B}_\kappa \end{bmatrix} \mathbf{u}_\partial, \\ \mathbf{y}_\partial &= \begin{bmatrix} \mathbf{0} & \mathbf{B}_\kappa^\top \end{bmatrix} \begin{pmatrix} \mathbf{e}_w \\ \mathbf{e}_\kappa \end{pmatrix}. \end{aligned} \quad (7.73)$$

1189 The matrice $\mathbf{D}_{-\partial_{xx}}$ is defined as $(i, \in \{1, n_w\}, m \in \{1, n_\kappa\})$

$$D_{-\partial_{xx}}^{im} = \left\langle \phi_w^i, -\partial_{xx} \phi_\kappa^m \right\rangle_{L^2(\Omega)}. \quad (7.74)$$

1190 The \mathbf{B}_κ is composed of four column vectors $\mathbf{B}_\kappa = [\mathbf{b}_\kappa^1 \mathbf{b}_\kappa^2 \mathbf{b}_\kappa^3 \mathbf{b}_\kappa^4]$

$$b_\kappa^{1,m} = -\phi_\kappa^m(0), \quad b_\kappa^{2,m} = \phi_\kappa^m(L), \quad b_\kappa^{3,m} = \partial_x \phi_\kappa^m(0), \quad b_\kappa^{4,m} = -\partial_x \phi_\kappa^m(L), \quad m \in \{1, n_\kappa\}. \quad (7.75)$$

1191 Both discretization require the use of Hermite polynomials to meet the regularity require-
1192 ment. Indeed, to lower the regularity requirement for the finite elements employed in the
1193 discretization, both lines can be integrated by parts. This will be discussed in Chap. 8.

1194 7.1.3.2 Kirchhoff plate

1195 The link between the energy and co-energy variables for the isotropic Kirchhoff model is the
1196 following (5.33)

$$\alpha_w = \rho h e_w, \quad \mathbf{A}_\kappa = \mathbf{C}_b \mathbf{E}_\kappa, \quad \text{where} \quad \mathbf{C}_b := \mathbf{D}_b^{-1} \quad (7.76)$$

1197 where ρ is the mass density, h the plate thickness and \mathbf{D}_b , the bending rigidity tensor, cf. Eq.
1198 (5.11). The bending compliance is given by

$$\mathbf{C}_b = \frac{12}{Eh^3} [(1 + \nu)(\cdot) - \nu \text{Tr}(\cdot) \mathbf{I}_{2 \times 2}]. \quad (7.77)$$

Given an open connected set $\Omega \subset \mathbb{R}^2$, the Kirchhoff plate model (5.42) in co-energy form controlled by forces and momenta is then expressed as

$$\begin{bmatrix} \rho h & 0 \\ 0 & \mathbf{C}_b \end{bmatrix} \frac{\partial}{\partial t} \begin{pmatrix} e_w \\ \mathbf{E}_\kappa \end{pmatrix} = \begin{bmatrix} 0 & -\text{div Div} \\ \text{Hess} & \mathbf{0} \end{bmatrix} \begin{pmatrix} e_w \\ \mathbf{E}_\kappa \end{pmatrix}, \quad \begin{aligned} e_w &\in H^2(\Omega), \\ \mathbf{E}_\kappa &\in H^{\text{div Div}}(\Omega, \mathbb{R}_{\text{sym}}^{2 \times 2}), \end{aligned} \quad (7.78a)$$

$$\mathbf{u}_\partial = \begin{bmatrix} 0 & \gamma_{nn,1} \\ 0 & \gamma_{nn} \end{bmatrix} \begin{pmatrix} e_w \\ \mathbf{E}_\kappa \end{pmatrix}, \quad \mathbf{u}_\partial \in \mathbb{R}^2, \quad (7.78b)$$

$$\mathbf{y}_\partial = \begin{bmatrix} \gamma_0 & 0 \\ \gamma_1 & 0 \end{bmatrix} \begin{pmatrix} e_w \\ \mathbf{E}_\kappa \end{pmatrix}, \quad \mathbf{y}_\partial \in \mathbb{R}^2, \quad (7.78c)$$

1199 We recall the expressions of the trace maps

$$\begin{aligned}\gamma_0 a &= a|_{\partial\Omega}, & \gamma_{nn,1} \mathbf{A} &= -\mathbf{n} \cdot \text{Div } \mathbf{A} - \partial_s(\mathbf{A} : (\mathbf{n} \otimes \mathbf{s}))|_{\partial\Omega}, \\ \gamma_1 a &= \partial_{\mathbf{n}} a|_{\partial\Omega}, & \gamma_{nn} \mathbf{A} &= \mathbf{A} : (\mathbf{n} \otimes \mathbf{n})|_{\partial\Omega}.\end{aligned}\quad (7.79)$$

1200 The Hilbert spaces here considered are $X_1 = L^2(\Omega)$, $X_2 = L^2(\Omega, \mathbb{R}_{\text{sym}}^{2 \times 2})$. The operators
1201 \mathcal{L} , $N_{\partial,1}$, $N_{\partial,2}$ are

$$\mathcal{L} = \text{Hess}, \quad N_{\partial,1} = \begin{bmatrix} \gamma_0 \\ \gamma_1 \end{bmatrix}, \quad N_{\partial,2} = \begin{bmatrix} \gamma_{nn,1} \\ \gamma_{nn} \end{bmatrix}. \quad (7.80)$$

1202 The energy rate from Eq. (5.39) equals $\dot{H} = \langle \mathbf{y}_{\partial}, \mathbf{u}_{\partial} \rangle_{L^2(\partial\Omega, \mathbb{R}^2)}$. Introducing the test
1203 functions $(v_w, \mathbf{V}_{\kappa})$ and integrating by parts twice the first line of (7.78a) one gets

$$\begin{aligned}\langle v_w, \rho h \partial_t e_w \rangle_{L^2(\Omega)} &= -\langle \text{Hess } v_w, \mathbf{E}_{\kappa} \rangle_{L^2(\Omega, \mathbb{R}_{\text{sym}}^{2 \times 2})} + \langle \gamma_0 v_w, u_{\partial,1} \rangle_{L^2(\partial\Omega)} + \langle \gamma_1 v_w, u_{\partial,2} \rangle_{L^2(\partial\Omega)}, \\ \langle \mathbf{V}_{\kappa}, \mathbf{C}_b \partial_t \mathbf{V}_{\kappa} \rangle_{L^2(\Omega, \mathbb{R}_{\text{sym}}^{2 \times 2})} &= \langle \mathbf{V}_{\kappa}, \text{Hess } e_w \rangle_{L^2(\Omega, \mathbb{R}_{\text{sym}}^{2 \times 2})}.\end{aligned}\quad (7.81)$$

1204 Introducing a Galerkin discretization for test and efforts functions

$$\begin{aligned}v_w &= \sum_{i=1}^{n_w} \phi_w^i v_w^i, & \mathbf{V}_{\kappa} &= \sum_{i=1}^{n_{\kappa}} \Phi_{\kappa}^i v_{\kappa}^i, & v_{\partial} &= \sum_{i=1}^{n_{\partial}} \phi_{\partial}^i v_{\partial}^i, & \mathbf{y}_{\partial} &= \sum_{i=1}^{n_{\partial}} \phi_{\partial}^i y_{\partial}^i. \\ e_w &= \sum_{i=1}^{n_w} \phi_w^i e_w^i, & \mathbf{E}_{\kappa} &= \sum_{i=1}^{n_{\kappa}} \Phi_{\kappa}^i e_{\kappa}^i, & \mathbf{u}_{\partial} &= \sum_{i=1}^{n_{\partial}} \phi_{\partial}^i u_{\partial}^i,\end{aligned}\quad (7.82)$$

1205 the following finite dimensional system is obtained

$$\begin{aligned}\begin{bmatrix} \mathbf{M}_{\rho h} & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_{\mathbf{C}_b} \end{bmatrix} \begin{pmatrix} \dot{\mathbf{e}}_w \\ \dot{\mathbf{e}}_{\kappa} \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & -\mathbf{D}_{\text{Hess}}^{\top} \\ \mathbf{D}_{\text{Hess}} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_w \\ \mathbf{e}_{\kappa} \end{pmatrix} + \begin{bmatrix} \mathbf{B}_w & \mathbf{B}_{\partial_n w} \\ \mathbf{0} & \mathbf{0} \end{bmatrix} \mathbf{u}_{\partial}, \\ \mathbf{M}_{\partial} \mathbf{y}_{\partial} &= \begin{bmatrix} \mathbf{B}_w^{\top} & \mathbf{0} \\ \mathbf{B}_{\partial_n w}^{\top} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_w \\ \mathbf{e}_{\kappa} \end{pmatrix}.\end{aligned}\quad (7.83)$$

1206 The matrices $\mathbf{M}_{\rho h}$, $\mathbf{M}_{\mathbf{C}_b}$, \mathbf{D}_{Hess} are defined as $(i, j \in \{1, n_w\}, m, n \in \{1, n_{\kappa}\})$

$$M_{\rho h}^{ij} = \langle \phi_w^i, \rho h \phi_w^j \rangle_{L^2(\Omega)}, \quad M_{\mathbf{C}_b}^{mn} = \langle \Phi_{\kappa}^m, \mathbf{C}_b \Phi_{\kappa}^n \rangle_{L^2(\Omega, \mathbb{R}_{\text{sym}}^{2 \times 2})}, \quad D_{\text{Hess}}^{mi} = \langle \Phi_{\kappa}^m, \text{Hess } \phi_w^i \rangle_{L^2(\Omega)}. \quad (7.84)$$

1207 Matrices \mathbf{B}_w , $\mathbf{B}_{\partial_n w}$ are given by

$$B_w^{il} = \langle \gamma_0 \phi_w^i, \phi_{\partial,1}^l \rangle_{L^2(\partial\Omega)}, \quad B_{\partial_n w}^{il} = \langle \gamma_1 \phi_w^i, \phi_{\partial,2}^l \rangle_{L^2(\partial\Omega)}, \quad l \in \{1, n_{\partial}\}. \quad (7.85)$$

1208 This kind of discretization requires H^2 conforming element. The construction of those is
1209 rather involved [AFS68, Bel69] and they are computationally expensive. Nevertheless, this
1210 kind of discretization is able to handle generic boundary conditions [GSV18]. For this reason,
1211 it is the most adapted for the pH framework.

To lower the regularity requirement for the finite elements many non conforming discretization have been proposed. The most employed is the Hellan-Herrmann-Johnson element [AB85, BR90]. However, this method does not handle generic non homogeneous boundary conditions. Given the unavailability of the boundary for interconnections, the modularity feature of pHs cannot be fully exploited.

Equivalently, the second line of Eq. (7.78a) can be integrated by parts twice to obtain a discretized system whose input are the linear velocity and the angular velocity at the boundary. However, while for the H^2 space conforming finite elements are available, for the $H^{\text{div Div}}$ no conforming finite elements have been proposed. This makes the discretization unfeasible.

7.1.3.3 Mindlin plate

Using Eqs. (5.22) and (5.24), the relation between co-energy and energy variables for the isotropic Mindlin plate is found to be

$$\begin{aligned} \alpha_w &= \rho h e_w, & \alpha_\theta &= I_\theta e_\theta, & I_\theta &:= \rho h^3/12, \\ \mathbf{A}_\kappa &= \mathbf{C}_b \mathbf{E}_\kappa, & \alpha_\gamma &= C_s e_\gamma, & C_s &:= 1/(kGh), \end{aligned} \tag{7.86}$$

where k is the shear correction factor, G the shear modulus. The other variables have the same meaning as in Sec. §7.1.3.2.

Control through forces and torques A pH representation in co-energy variables with known forces and momenta at the boundary is given by the system

$$\begin{bmatrix} \rho h & 0 & 0 & 0 \\ \mathbf{0} & I_\theta & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{C}_b & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{0} & C_s \end{bmatrix} \frac{\partial}{\partial t} \begin{pmatrix} e \\ \mathbf{e}_\theta \\ \mathbf{E}_\kappa \\ \mathbf{e}_\gamma \end{pmatrix} = \begin{bmatrix} 0 & 0 & 0 & \text{div} \\ \mathbf{0} & \mathbf{0} & \text{Div} & \mathbf{I}_{2 \times 2} \\ \mathbf{0} & \text{Grad} & \mathbf{0} & \mathbf{0} \\ \text{grad} & -\mathbf{I}_{2 \times 2} & \mathbf{0} & \mathbf{0} \end{bmatrix} \begin{pmatrix} e_w \\ \mathbf{e}_\theta \\ \mathbf{E}_\kappa \\ \mathbf{e}_\gamma \end{pmatrix}, \quad \begin{aligned} e_w &\in H^1(\Omega), \\ \mathbf{e}_\theta &\in H^{\text{Grad}}(\Omega, \mathbb{R}^2), \\ \mathbf{E}_\kappa &\in H^{\text{Div}}(\Omega, \mathbb{R}_{\text{sym}}^{2 \times 2}), \\ \mathbf{e}_\gamma &\in H^{\text{div}}(\Omega, \mathbb{R}^2), \end{aligned} \quad (7.87a)$$

$$\mathbf{u}_\partial = \begin{bmatrix} 0 & 0 & 0 & \gamma_n \\ 0 & 0 & \gamma_{nn} & 0 \\ 0 & 0 & \gamma_{ns} & 0 \end{bmatrix} \begin{pmatrix} e_w \\ \mathbf{e}_\theta \\ \mathbf{E}_\kappa \\ \mathbf{e}_\gamma \end{pmatrix}, \quad \mathbf{u}_\partial \in \mathbb{R}^3, \quad (7.87b)$$

$$\mathbf{y}_\partial = \begin{bmatrix} \gamma_0 & 0 & 0 & 0 \\ 0 & \gamma_n & 0 & 0 \\ 0 & \gamma_s & 0 & 0 \end{bmatrix} \begin{pmatrix} e_w \\ \mathbf{e}_\theta \\ \mathbf{E}_\kappa \\ \mathbf{e}_\gamma \end{pmatrix}, \quad \mathbf{y}_\partial \in \mathbb{R}^3. \quad (7.87c)$$

1231 The trace operators are defined as

$$\begin{aligned} \gamma_0 a &= a|_{\partial\Omega}, & \gamma_n \mathbf{a} &= \mathbf{a} \cdot \mathbf{n}|_{\partial\Omega}, & \gamma_{nn} \mathbf{A} &= \mathbf{A} : (\mathbf{n} \otimes \mathbf{n})|_{\partial\Omega}, \\ \gamma_s \mathbf{a} &= \mathbf{a} \cdot \mathbf{s}|_{\partial\Omega}, & \gamma_{ns} \mathbf{A} &= \mathbf{A} : (\mathbf{n} \otimes \mathbf{s})|_{\partial\Omega}. \end{aligned} \quad (7.88)$$

1232 For this example, the Hilbert spaces under consideration are $X_1 = L^2(\Omega) \times L^2(\Omega, \mathbb{R}^2)$, $X_2 =$
 1233 $L^2(\Omega, \mathbb{R}_{\text{sym}}^{2 \times 2}) \times L^2(\Omega, \mathbb{R}^2)$. The \mathbf{L} , \mathcal{L} , $\mathcal{N}_{\partial,1}$, $\mathcal{N}_{\partial,1}$ operators are

$$\mathbf{L} = \begin{bmatrix} \mathbf{0} & \mathbf{0} \\ \mathbf{0} & -\mathbf{I}_{2 \times 2} \end{bmatrix}, \quad \mathcal{L} = \begin{bmatrix} \mathbf{0} & \text{Grad} \\ \text{grad} & \mathbf{0} \end{bmatrix}, \quad \mathcal{N}_{\partial,1} = \begin{bmatrix} \gamma_0 & 0 \\ 0 & \gamma_n \\ 0 & \gamma_s \end{bmatrix}, \quad \mathcal{N}_{\partial,2} = \begin{bmatrix} 0 & \gamma_n \\ \gamma_{nn} & 0 \\ \gamma_{ns} & 0 \end{bmatrix}. \quad (7.89)$$

1234 The energy rate is retrieved from Eq. (5.26) $\dot{H} = \langle \mathbf{y}_\partial, \mathbf{u}_\partial \rangle_{L^2(\partial\Omega, \mathbb{R}^2)}$. Introducing the test
 1235 functions $(v_w, \mathbf{v}_\theta, \mathbf{V}_\kappa, \mathbf{v}_\gamma)$ and integrating by parts the first two lines of (7.87a) one gets

$$\begin{aligned} \langle v_w, \rho h \partial_t e_w \rangle_{L^2(\Omega)} &= -\langle \text{grad } v_w, \mathbf{e}_\gamma \rangle_{L^2(\Omega, \mathbb{R}^2)} + \langle \gamma_0 v_w, u_{\partial,1} \rangle_{L^2(\partial\Omega)}, \\ \langle \mathbf{v}_\theta, I_\theta \partial_t \mathbf{e}_\theta \rangle_{L^2(\Omega, \mathbb{R}^2)} &= -\langle \text{Grad } \mathbf{v}_\theta, \mathbf{E}_\kappa \rangle_{L^2(\Omega, \mathbb{R}_{\text{sym}}^{2 \times 2})} + \langle \mathbf{v}_\theta, \mathbf{e}_\gamma \rangle_{L^2(\Omega)} + \langle \gamma_0 \mathbf{v}_\theta, \gamma_n \mathbf{E}_\kappa \rangle_{L^2(\partial\Omega, \mathbb{R}^2)}, \\ \langle \mathbf{V}_\kappa, \mathbf{C}_b \partial_t \mathbf{E}_\kappa \rangle_{L^2(\Omega, \mathbb{R}_{\text{sym}}^{2 \times 2})} &= \langle \mathbf{V}_\kappa, \text{Grad } \mathbf{e}_\theta \rangle_{L^2(\Omega, \mathbb{R}_{\text{sym}}^{2 \times 2})}, \\ \langle \mathbf{v}_\gamma, C_s \partial_t \mathbf{e}_\gamma \rangle_{L^2(\Omega, \mathbb{R}^2)} &= \langle \mathbf{v}_\gamma, \text{grad } e_w \rangle_{L^2(\Omega, \mathbb{R}^2)} - \langle \mathbf{v}_\gamma, \mathbf{e}_\theta \rangle_{L^2(\Omega, \mathbb{R}^2)}. \end{aligned} \quad (7.90)$$

1236 The term $\langle \gamma_0 \mathbf{v}_\theta, \mathbf{u}_{\partial,2} \rangle_{L^2(\partial\Omega, \mathbb{R}^2)}$ can be decomposed in its tangential and normal components

$$\langle \gamma_0 \mathbf{v}_\theta, \gamma_n \mathbf{E}_\kappa \rangle_{L^2(\partial\Omega, \mathbb{R}^2)} = \langle \gamma_n \mathbf{v}_\theta, u_{\partial,2} \rangle_{L^2(\partial\Omega)} + \langle \gamma_s \mathbf{v}_\theta, u_{\partial,3} \rangle_{L^2(\partial\Omega)} \quad (7.91)$$

1237 Introducing a Galerkin discretization for test and efforts functions

$$\begin{aligned}
 v_w &= \sum_{i=1}^{n_w} \phi_w^i v_w^i, & v_\theta &= \sum_{i=1}^{n_\theta} \phi_\theta^i v_\theta^i, & V_\kappa &= \sum_{i=1}^{n_\kappa} \Phi_\kappa^i v_\kappa^i, & v_\gamma &= \sum_{i=1}^{n_\gamma} \phi_\gamma^i v_\gamma^i, & v_\partial &= \sum_{i=1}^{n_\partial} \phi_\partial^i v_\partial^i, \\
 e_w &= \sum_{i=1}^{n_w} \phi_w^i e_w^i, & e_\theta &= \sum_{i=1}^{n_\theta} \phi_\theta^i e_\theta^i, & E_\kappa &= \sum_{i=1}^{n_\kappa} \Phi_\kappa^i e_\kappa^i, & e_\gamma &= \sum_{i=1}^{n_\gamma} \phi_\gamma^i e_\gamma^i, & u_\partial &= \sum_{i=1}^{n_\partial} \phi_\partial^i u_\partial^i, \\
 & & & & & & & & y_\partial &= \sum_{i=1}^{n_\partial} \phi_\partial^i y_\partial^i.
 \end{aligned} \tag{7.92}$$

1238 the following finite dimensional system is obtained

$$\begin{aligned}
 \text{Diag} \begin{bmatrix} \mathbf{M}_{\rho h} \\ \mathbf{M}_{I_\theta} \\ \mathbf{M}_{\mathbf{C}_b} \\ \mathbf{M}_{C_s} \end{bmatrix} \begin{pmatrix} \dot{\mathbf{e}}_w \\ \dot{\mathbf{e}}_\theta \\ \dot{\mathbf{e}}_\kappa \\ \dot{\mathbf{e}}_\gamma \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & \mathbf{0} & \mathbf{0} & -\mathbf{D}_{\text{grad}}^\top \\ \mathbf{0} & \mathbf{0} & -\mathbf{D}_{\text{Grad}}^\top & -\mathbf{D}_0^\top \\ \mathbf{0} & \mathbf{D}_{\text{Grad}} & \mathbf{0} & \mathbf{0} \\ \mathbf{D}_{\text{grad}} & \mathbf{D}_0 & \mathbf{0} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_w \\ \mathbf{e}_\theta \\ \mathbf{e}_\kappa \\ \mathbf{e}_\gamma \end{pmatrix} + \begin{bmatrix} \mathbf{B}_w & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \mathbf{B}_{\theta_n} & \mathbf{B}_{\theta_s} \\ \mathbf{0} & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{0} \end{bmatrix} \mathbf{u}_\partial, \\
 \mathbf{M}_{\partial} \mathbf{y}_\partial &= \begin{bmatrix} \mathbf{B}_w^\top & \mathbf{0} & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \mathbf{B}_{\theta_n}^\top & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{B}_{\theta_s}^\top & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_w \\ \mathbf{e}_\theta \\ \mathbf{e}_\kappa \\ \mathbf{e}_\gamma \end{pmatrix}.
 \end{aligned} \tag{7.93}$$

1239 The notation Diag denotes a block diagonal matrix. The mass matrices $\mathbf{M}_{\rho h}$, \mathbf{M}_{I_θ} , $\mathbf{M}_{\mathbf{C}_b}$, \mathbf{M}_{C_s}
 1240 are computed as

$$\begin{aligned}
 M_{\rho h}^{ij} &= \langle \phi_w^i, \rho h \phi_w^j \rangle_{L^2(\Omega)}, & M_{\mathbf{C}_b}^{pq} &= \langle \Phi_\kappa^p, \mathbf{C}_b \Phi_\kappa^q \rangle_{L^2(\Omega, \mathbb{R}_{\text{sym}}^{2 \times 2})}, \\
 M_{I_\theta}^{mn} &= \langle \phi_\kappa^m, I_\theta \phi_\kappa^n \rangle_{L^2(\Omega, \mathbb{R}^2)}, & M_{C_s}^{rs} &= \langle \phi_\gamma^r, C_s \phi_\gamma^s \rangle_{L^2(\Omega, \mathbb{R}^2)},
 \end{aligned} \tag{7.94}$$

1241 where $i, j \in \{1, n_w\}$, $m, n \in \{1, n_\theta\}$, $p, q \in \{1, n_\kappa\}$, $r, s \in \{1, n_\gamma\}$. Matrices \mathbf{D}_{grad} , \mathbf{D}_{Grad} , \mathbf{D}_0
 1242 assume the form

$$\begin{aligned}
 D_{\text{grad}}^{rj} &= \langle \phi_\gamma^r, \text{grad } \phi_w^j \rangle_{L^2(\Omega, \mathbb{R}^2)}, & D_0^{rn} &= -\langle \phi_\gamma^r, \phi_\theta^n \rangle_{L^2(\Omega, \mathbb{R}^2)}, \\
 D_{\text{Grad}}^{pn} &= \langle \Phi_\kappa^p, \text{Grad } \phi_\theta^n \rangle_{L^2(\Omega, \mathbb{R}_{\text{sym}}^{2 \times 2})},
 \end{aligned} \tag{7.95}$$

1243 Matrix \mathbf{B}_w , \mathbf{B}_{θ_n} , \mathbf{B}_{θ_s} are computed as ($l \in \{1, n_\partial\}$)

$$B_w^{il} = \langle \gamma_0 \phi_w^i, \phi_{\partial,1}^l \rangle_{L^2(\partial\Omega)}, \quad B_{\theta_n}^{ml} = \langle \gamma_n \phi_\theta^m, \phi_{\partial,2}^l \rangle_{L^2(\partial\Omega)}, \quad B_{\theta_s}^{ml} = \langle \gamma_s \phi_\theta^m, \phi_{\partial,3}^l \rangle_{L^2(\partial\Omega)}. \tag{7.96}$$

Control through linear and angular velocities If instead the opposite causality is considered, the continuous system read

$$\begin{bmatrix} \rho h & 0 & 0 & 0 \\ \mathbf{0} & I_\theta & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{C}_b & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{0} & C_s \end{bmatrix} \frac{\partial}{\partial t} \begin{pmatrix} e \\ \mathbf{e}_\theta \\ \mathbf{E}_\kappa \\ \mathbf{e}_\gamma \end{pmatrix} = \begin{bmatrix} 0 & 0 & 0 & \text{div} \\ \mathbf{0} & \mathbf{0} & \text{Div} & \mathbf{I}_{2 \times 2} \\ \mathbf{0} & \text{Grad} & \mathbf{0} & \mathbf{0} \\ \text{grad} & -\mathbf{I}_{2 \times 2} & \mathbf{0} & \mathbf{0} \end{bmatrix} \begin{pmatrix} e_w \\ \mathbf{e}_\theta \\ \mathbf{E}_\kappa \\ \mathbf{e}_\gamma \end{pmatrix}, \quad (7.97a)$$

$$\mathbf{u}_\partial = \begin{bmatrix} \gamma_0 & 0 & 0 & 0 \\ 0 & \gamma_n & 0 & 0 \\ 0 & \gamma_s & 0 & 0 \end{bmatrix} \begin{pmatrix} e_w \\ \mathbf{e}_\theta \\ \mathbf{E}_\kappa \\ \mathbf{e}_\gamma \end{pmatrix}, \quad \mathbf{u}_\partial \in \mathbb{R}^3, \quad (7.97b)$$

$$\mathbf{y}_\partial = \begin{bmatrix} 0 & 0 & 0 & \gamma_n \\ 0 & 0 & \gamma_{nn} & 0 \\ 0 & 0 & \gamma_{ns} & 0 \end{bmatrix} \begin{pmatrix} e_w \\ \mathbf{e}_\theta \\ \mathbf{E}_\kappa \\ \mathbf{e}_\gamma \end{pmatrix}, \quad \mathbf{y}_\partial \in \mathbb{R}^3. \quad (7.97c)$$

1244 integrating by parts the last two lines of (7.97a) one gets

$$\begin{aligned} \langle v_w, \rho h \partial_t e_w \rangle_{L^2(\Omega)} &= \langle v_w, \text{div } \mathbf{e}_\gamma \rangle_{L^2(\Omega, \mathbb{R}^2)}, \\ \langle \mathbf{v}_\theta, I_\theta \partial_t \mathbf{e}_\theta \rangle_{L^2(\Omega, \mathbb{R}^2)} &= \langle \mathbf{v}_\theta, \text{Div } \mathbf{E}_\kappa \rangle_{L^2(\Omega, \mathbb{R}^2)} + \langle \mathbf{v}_\theta, \mathbf{e}_\gamma \rangle_{L^2(\Omega)}, \\ \langle \mathbf{V}_\kappa, \mathbf{C}_b \partial_t \mathbf{E}_\kappa \rangle_{L^2(\Omega, \mathbb{R}_{\text{sym}}^{2 \times 2})} &= -\langle \text{Div } \mathbf{V}_\kappa, \mathbf{e}_\theta \rangle_{L^2(\Omega, \mathbb{R}^2)} + \langle \gamma_n \mathbf{V}_\kappa, \gamma_0 \mathbf{e}_\theta \rangle_{L^2(\partial\Omega, \mathbb{R}^2)}, \\ \langle \mathbf{v}_\gamma, C_s \partial_t \mathbf{e}_\gamma \rangle_{L^2(\Omega, \mathbb{R}^2)} &= -\langle \text{div } \mathbf{v}_\gamma, e_w \rangle_{L^2(\Omega)} - \langle \mathbf{v}_\gamma, \mathbf{e}_\theta \rangle_{L^2(\Omega, \mathbb{R}^2)} + \langle \gamma_0 v_w, u_{\partial,1} \rangle_{L^2(\partial\Omega)}. \end{aligned} \quad (7.98)$$

1245 The term $\langle \gamma_n \mathbf{V}_\kappa, \gamma_0 \mathbf{e}_\theta \rangle_{L^2(\partial\Omega, \mathbb{R}^2)}$ can be decomposed in its tangential and normal components

$$\langle \gamma_n \mathbf{V}_\kappa, \gamma_0 \mathbf{e}_\theta \rangle_{L^2(\partial\Omega, \mathbb{R}^2)} = \langle \gamma_{nn} \mathbf{V}_\kappa, u_{\partial,2} \rangle_{L^2(\partial\Omega)} + \langle \gamma_{ns} \mathbf{V}_\kappa, u_{\partial,3} \rangle_{L^2(\partial\Omega)}. \quad (7.99)$$

1246 Plugging approximation (7.92) into this system, one computes

$$\begin{aligned} \text{Diag} \begin{bmatrix} \mathbf{M}_{\rho h} \\ \mathbf{M}_{I_\theta} \\ \mathbf{M}_{\mathbf{C}_b} \\ \mathbf{M}_{C_s} \end{bmatrix} \begin{pmatrix} \dot{e}_w \\ \dot{\mathbf{e}}_\theta \\ \dot{\mathbf{E}}_\kappa \\ \dot{\mathbf{e}}_\gamma \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & \mathbf{0} & \mathbf{0} & \mathbf{D}_{\text{div}} \\ \mathbf{0} & \mathbf{0} & \mathbf{D}_{\text{Div}} & -\mathbf{D}_0^\top \\ \mathbf{0} & -\mathbf{D}_{\text{Div}}^\top & \mathbf{0} & \mathbf{0} \\ -\mathbf{D}_{\text{div}}^\top & \mathbf{D}_0 & \mathbf{0} & \mathbf{0} \end{bmatrix} \begin{pmatrix} e_w \\ \mathbf{e}_\theta \\ \mathbf{e}_\kappa \\ \mathbf{e}_\gamma \end{pmatrix} + \begin{bmatrix} \mathbf{0} & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{0} \\ \mathbf{0} & \mathbf{B}_{M_{nn}} & \mathbf{B}_{M_{ns}} \\ \mathbf{B}_{q_n} & \mathbf{0} & \mathbf{0} \end{bmatrix} \mathbf{u}_\partial, \\ \mathbf{M}_{\partial} \mathbf{y}_\partial &= \begin{bmatrix} \mathbf{0} & \mathbf{0} & \mathbf{0} & \mathbf{B}_{q_n}^\top \\ \mathbf{0} & \mathbf{0} & \mathbf{B}_{M_{nn}}^\top & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{B}_{M_{ns}}^\top & \mathbf{0} \end{bmatrix} \begin{pmatrix} e_w \\ \mathbf{e}_\theta \\ \mathbf{e}_\kappa \\ \mathbf{e}_\gamma \end{pmatrix}. \end{aligned} \quad (7.100)$$

1247 Matrices \mathbf{D}_{div} , \mathbf{D}_{Div} assume the form ($i, j \in \{1, n_w\}$, $m, n \in \{1, n_\theta\}$, $p, q \in \{1, n_\kappa\}$, $r, s \in$
1248 $\{1, n_\gamma\}$)

$$D_{\text{div}}^{is} = \langle \phi_w^i, \text{div } \phi_\gamma^s \rangle_{L^2(\Omega)}, \quad D_{\text{Div}}^{mq} = \langle \phi_\theta^m, \text{Div } \Phi_\kappa^q \rangle_{L^2(\Omega, \mathbb{R}^2)}. \quad (7.101)$$

Matrix \mathbf{B}_{q_n} , $\mathbf{B}_{M_{nn}}$, $\mathbf{B}_{M_{ns}}$ are computed as ($l \in \{1, n_\partial\}$)

$$B_{q_n}^{rl} = \langle \gamma_n \phi_\gamma^r, \phi_{\partial,1}^l \rangle_{L^2(\partial\Omega)}, \quad B_{M_{nn}}^{pl} = \langle \gamma_{nn} \Phi_\kappa^p, \phi_{\partial,2}^l \rangle_{L^2(\partial\Omega)}, \quad B_{M_{ns}}^{pl} = \langle \gamma_{ns} \Phi_\kappa^p, \phi_{\partial,3}^l \rangle_{L^2(\partial\Omega)}. \quad (7.102)$$

This finite dimensional system represents a purely mixed discretization of the problem and is really close to the plane elasticity system. Conforming finite elements for the plane elasticity system on simplicial meshes have been constructed in [AW02]. The resulting element is rather cumbersome and computationally expensive as the stress tensor has at least 24 degrees of freedom on a triangle. For this reason, many finite element discretization imposes the symmetry of the stress tensor weakly [AFW07]. To actually implement the discretization, in Chap. 8 the Mindlin plate problem is going to be reformulated so that the momenta tensor is only weakly symmetric.

7.2 Mixed boundary conditions

In this section the assumption of uniform boundary condition 2 is removed. This allows considering general non homogeneous boundary conditions. The discretization of Stokes-Dirac structure under mixed causality has been already treated in [KML18]. However, to satisfy the power balance at a discrete level, some additional parameters are introduced. This makes the employment of this methodology not simple and really dependent on the considered application. Furthermore, elasticity models do not fall within the required assumptions.

We propose here two methodologies to tackle mixed boundary conditions within the Partitioned Finite Element Method. One introduces Lagrange multipliers, and therefore algebraic constraints, to enforce the mixed causality. Finite dimensional differential algebraic port-Hamiltonian systems (pHDAE) have been introduced in [BMXZ18] for linear systems and in [MM19] for non linear systems. This enriched description share all the crucial features of ordinary pHs, but easily account for algebraic constraints, time-dependent transformations and explicit dependence on time in the Hamiltonian. The other method employs a domain decomposition technique to interconnect systems with different causalities.

We will limit the illustration to the linear case. The open connected set $\Omega \subset \mathbb{R}^d$, $d = \{1, 2, 3\}$, with Lipschitz boundary $\partial\Omega$ represent the spatial domain. The boundary is split into two partition $\partial\Omega = \bar{\Gamma}_1 \cup \bar{\Gamma}_2$, $\Gamma_1 \cap \Gamma_2 = \{\emptyset\}$. For ease of presentation, Γ_1 , Γ_2 are considered to be connected, cf. Fig. 7.1. Disconnected sets can be handled as well. For scalars $a_{\partial,*}, b_{\partial,*} \in L^2(\Gamma_*)$ and vectors $\mathbf{a}_{\partial,*}, \mathbf{b}_{\partial,*} \in L^2(\Gamma_*, \mathbb{R}^m)$ defined on the boundary partition Γ_* the inner product is defined as

$$\langle a_{\partial,*}, b_{\partial,*} \rangle_{L^2(\Gamma_*)} = \int_{\Gamma_*} a_{\partial,*} b_{\partial,*} \, d\Gamma_*, \quad \langle \mathbf{a}_{\partial,*}, \mathbf{b}_{\partial,*} \rangle_{L^2(\Gamma_*, \mathbb{R}^m)} = \int_{\Gamma_*} \mathbf{a}_{\partial,*} \cdot \mathbf{b}_{\partial,*} \, d\Gamma_*. \quad (7.103)$$

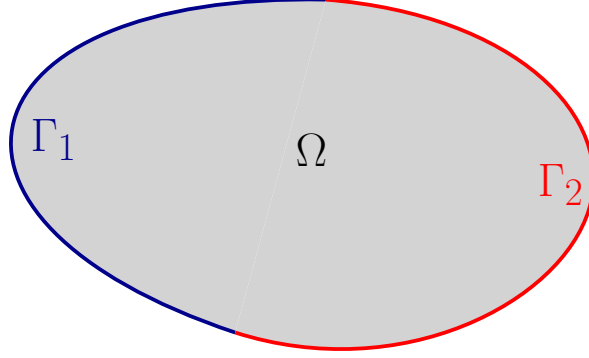


Figure 7.1: Partition of boundary into two connected sets.

Consider now the following boundary control linear pH system in co-energy form

$$\begin{bmatrix} \mathcal{M}_1 & 0 \\ 0 & \mathcal{M}_2 \end{bmatrix} \partial_t \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix} = \begin{bmatrix} 0 & -\mathbf{L}^\top - \mathcal{L}^* \\ \mathbf{L} + \mathcal{L} & 0 \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix}, \quad \begin{matrix} \mathbf{e}_1 \in H^\mathcal{L}, \\ \mathbf{e}_2 \in H^{-\mathcal{L}^*}. \end{matrix} \quad (7.104a)$$

$$\begin{pmatrix} \mathbf{u}_{\partial,1} \\ \mathbf{u}_{\partial,2} \end{pmatrix} = \begin{bmatrix} \mathcal{N}_{\partial,1}^{\Gamma_1} & 0 \\ 0 & \mathcal{N}_{\partial,2}^{\Gamma_2} \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix}, \quad \begin{matrix} \mathbf{u}_{\partial,1} \in \mathbb{R}^m, \\ \mathbf{u}_{\partial,2} \in \mathbb{R}^m, \end{matrix} \quad (7.104b)$$

$$\begin{pmatrix} \mathbf{y}_{\partial,1} \\ \mathbf{y}_{\partial,2} \end{pmatrix} = \begin{bmatrix} 0 & \mathcal{N}_{\partial,2}^{\Gamma_1} \\ \mathcal{N}_{\partial,1}^{\Gamma_2} & 0 \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix}, \quad \begin{matrix} \mathbf{y}_{\partial,1} \in \mathbb{R}^m, \\ \mathbf{y}_{\partial,2} \in \mathbb{R}^m. \end{matrix} \quad (7.104c)$$

1281 The operator $\mathcal{N}_{\partial,*}^{\Gamma_\circ}$ with $*, \circ \in \{1, 2\}$ represent the restriction of operator $\mathcal{N}_{\partial,*}$, defined in
 1282 Eq. (7.9), over the subset Γ_\circ . The boundary inputs and output are now vectors \mathbb{R}^{2m} . This
 1283 does not mean that the boundary conditions have been doubled, but only that the components
 1284 of $\mathbf{u}_\partial, \mathbf{y}_\partial$ are only defined on the subsets Γ_1, Γ_2 of the overall boundary. Given the additive
 1285 property of the integral, it is possible to write

$$\begin{aligned} \langle \mathcal{N}_{\partial,1} \mathbf{e}_1, \mathcal{N}_{\partial,2} \mathbf{e}_2 \rangle_{L^2(\partial\Omega)} &= \langle \mathcal{N}_{\partial,1}^{\Gamma_1} \mathbf{e}_1, \mathcal{N}_{\partial,2}^{\Gamma_1} \mathbf{e}_2 \rangle_{L^2(\Gamma_1)} + \langle \mathcal{N}_{\partial,1}^{\Gamma_2} \mathbf{e}_1, \mathcal{N}_{\partial,2}^{\Gamma_2} \mathbf{e}_2 \rangle_{L^2(\Gamma_2)}, \\ &= \langle \mathbf{u}_{\partial,1}, \mathbf{y}_{\partial,1} \rangle_{L^2(\Gamma_1)} + \langle \mathbf{y}_{\partial,2}, \mathbf{u}_{\partial,2} \rangle_{L^2(\Gamma_2)}. \end{aligned} \quad (7.105)$$

1286 The continuous power balance is obtained using Eqs. (7.51) and (7.105)

$$\dot{H} = \langle \mathbf{u}_{\partial,1}, \mathbf{y}_{\partial,1} \rangle_{L^2(\Gamma_1)} + \langle \mathbf{y}_{\partial,2}, \mathbf{u}_{\partial,2} \rangle_{L^2(\Gamma_2)}. \quad (7.106)$$

1287 7.2.1 Solution using Lagrange multipliers

1288 This solution consists in introducing a Lagrange multiplier for the boundary control that does
 1289 not arise in the weak form. To illustrate the idea, consider again the weak form 7.52 (obtained
 1290 by integration by parts of the $-\mathcal{L}^*$ partitioned) of Sys. 7.104

$$\begin{aligned}\langle \mathbf{v}_1, \mathcal{M}_1 \partial_t \mathbf{e}_1 \rangle_{X_1} &= -\langle \mathbf{v}_1, \mathbf{L}^\top \mathbf{e}_2 \rangle_{X_1} - \langle \mathcal{L} \mathbf{v}_1, \mathbf{e}_2 \rangle_{X_2} + \langle \mathcal{N}_{\partial,1} \mathbf{v}_1, \mathcal{N}_{\partial,2} \mathbf{e}_2 \rangle_{L^2(\partial\Omega)}, \\ \langle \mathbf{v}_2, \mathcal{M}_2 \partial_t \mathbf{e}_2 \rangle_{X_2} &= \langle \mathbf{v}_2, \mathbf{L} \mathbf{e}_1 \rangle_{X_2} + \langle \mathbf{v}_2, \mathcal{L} \mathbf{e}_1 \rangle_{X_2},\end{aligned}\quad (7.107)$$

1291 The term $\langle \mathcal{N}_{\partial,1} \mathbf{v}_1, \mathcal{N}_{\partial,2} \mathbf{e}_2 \rangle_{L^2(\partial\Omega)}$ can be split into the two boundary contributions, as in Eq.
1292 (7.105). The variable $\mathbf{y}_{\partial,1}$ plays here the role of a Lagrange multiplier $\mathbf{y}_{\partial,1} = \boldsymbol{\lambda}_{\partial,1}$

$$\begin{aligned}\langle \mathcal{N}_{\partial,1} \mathbf{v}_1, \mathcal{N}_{\partial,2} \mathbf{e}_2 \rangle_{L^2(\partial\Omega)} &= \langle \mathcal{N}_{\partial,1}^{\Gamma_1} \mathbf{v}_1, \mathcal{N}_{\partial,2}^{\Gamma_1} \mathbf{e}_2 \rangle_{L^2(\Gamma_1)} + \langle \mathcal{N}_{\partial,1}^{\Gamma_2} \mathbf{v}_1, \mathcal{N}_{\partial,2}^{\Gamma_2} \mathbf{e}_2 \rangle_{L^2(\Gamma_2)}, \\ &= \langle \mathcal{N}_{\partial,1}^{\Gamma_1} \mathbf{v}_1, \mathbf{y}_{\partial,1} \rangle_{L^2(\Gamma_1)} + \langle \mathcal{N}_{\partial,1}^{\Gamma_2} \mathbf{v}_1, \mathbf{u}_{\partial,2} \rangle_{L^2(\Gamma_2)}, \\ &= \langle \mathcal{N}_{\partial,1}^{\Gamma_1} \mathbf{v}_1, \boldsymbol{\lambda}_{\partial,1} \rangle_{L^2(\Gamma_1)} + \langle \mathcal{N}_{\partial,1}^{\Gamma_2} \mathbf{v}_1, \mathbf{u}_{\partial,2} \rangle_{L^2(\Gamma_2)},\end{aligned}\quad (7.108)$$

1293 If test function $\mathbf{v}_{\partial,1}, \mathbf{v}_{\partial,2} \in L^2(\Gamma_1) \times L^2(\Gamma_2)$ are introduced, the input and outputs definitions

$$\mathbf{u}_{\partial,1} = \mathcal{N}_{\partial,1}^{\Gamma_1} \mathbf{e}_1, \quad \mathbf{y}_{\partial,1} = \boldsymbol{\lambda}_{\partial,1}, \quad \mathbf{y}_{\partial,2} = \mathcal{N}_{\partial,1}^{\Gamma_2} \mathbf{e}_1 \quad (7.109)$$

1294 can be put into weak form to obtain

$$\begin{aligned}\langle \mathbf{v}_{\partial,1}, \mathbf{u}_{\partial,1} \rangle_{L^2(\Gamma_1)} &= \langle \mathbf{v}_{\partial,1}, \mathcal{N}_{\partial,1}^{\Gamma_1} \mathbf{e}_1 \rangle_{L^2(\Gamma_1)}, \\ \langle \mathbf{v}_{\partial,1}, \mathbf{y}_{\partial,1} \rangle_{L^2(\Gamma_1)} &= \langle \mathbf{v}_{\partial,1}, \boldsymbol{\lambda}_{\partial,1} \rangle_{L^2(\Gamma_1)}, \\ \langle \mathbf{v}_{\partial,2}, \mathbf{y}_{\partial,2} \rangle_{L^2(\Gamma_1)} &= \langle \mathbf{v}_{\partial,2}, \mathcal{N}_{\partial,1}^{\Gamma_2} \mathbf{e}_1 \rangle_{L^2(\Gamma_1)},\end{aligned}\quad (7.110)$$

1295 As usual, a Galerkin approximation is introduced

$$\begin{aligned}\mathbf{v}_1 &\approx \sum_{i=1}^{n_1} \phi_1^i(\mathbf{x}) v_1^i, & \mathbf{e}_1 &\approx \sum_{i=1}^{n_1} \phi_1^i(\mathbf{x}) e_1^i(t), & \Delta_{\partial,1} &\approx \sum_{i=1}^{n_{\partial,1}} \phi_{\partial,1}^i(\mathbf{s}_1) \Delta_{\partial,1}^i, & \mathbf{s}_1 &\in \Gamma_1, \\ \mathbf{v}_2 &\approx \sum_{i=1}^{n_2} \phi_2^i(\mathbf{x}) v_2^i, & \mathbf{e}_2 &\approx \sum_{i=1}^{n_2} \phi_2^i(\mathbf{x}) e_2^i(t), & \square_{\partial,2} &\approx \sum_{i=1}^{n_{\partial,2}} \phi_{\partial,2}^i(\mathbf{s}_2) \square_{\partial,2}^i(t), & \mathbf{s}_2 &\in \Gamma_2.\end{aligned}\quad (7.111)$$

1296 where Δ stays for v, u, y, λ and \square for v, u, y . Replacing the approximation 7.111 into Eqs.
1297 7.107, 7.108, 7.110, the following system differential-algebraic system is obtained

$$\begin{aligned}\text{Diag} \begin{bmatrix} \mathbf{M}_{\mathcal{M}_1} \\ \mathbf{M}_{\mathcal{M}_2} \\ \mathbf{0} \end{bmatrix} \begin{pmatrix} \dot{\mathbf{e}}_1 \\ \dot{\mathbf{e}}_2 \\ \dot{\boldsymbol{\lambda}}_{\partial,1} \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & -\mathbf{D}_0^\top - \mathbf{D}_{\mathcal{L}}^\top & \mathbf{B}_{1,\Gamma_1} \\ \mathbf{D}_0 + \mathbf{D}_{\mathcal{L}} & \mathbf{0} & \mathbf{0} \\ -\mathbf{B}_{1,\Gamma_1}^\top & \mathbf{0} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \\ \boldsymbol{\lambda}_{\partial,1} \end{pmatrix} + \begin{bmatrix} \mathbf{0} & \mathbf{B}_{1,\Gamma_2} \\ \mathbf{0} & \mathbf{0} \\ \mathbf{M}_{\partial,1} & \mathbf{0} \end{bmatrix} \begin{bmatrix} \mathbf{u}_{\partial,1} \\ \mathbf{u}_{\partial,2} \end{bmatrix}, \\ \begin{bmatrix} \mathbf{M}_{\partial,1} & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_{\partial,2} \end{bmatrix} \begin{pmatrix} \mathbf{y}_{\partial,1} \\ \mathbf{y}_{\partial,2} \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & \mathbf{0} & \mathbf{M}_{\partial,1} \\ \mathbf{B}_{1,\Gamma_2}^\top & \mathbf{0} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \\ \boldsymbol{\lambda}_{\partial,1} \end{pmatrix}.\end{aligned}\quad (7.112)$$

1298 Apart for matrices $\mathbf{M}_{\partial,1}, \mathbf{M}_{\partial,2}, \mathbf{B}_{1,\Gamma_1}, \mathbf{B}_{1,\Gamma_2}$,

$$\begin{aligned} M_{\partial,1}^{lk} &= \langle \phi_{\partial,1}^l, \phi_{\partial,1}^k \rangle_{L^2(\Gamma_1)}, \quad (l, k) \in \{1, n_{\partial,1}\}, & B_{1,\Gamma_1}^{ik} &= \langle \phi_1^i, \phi_{\partial,1}^k \rangle_{L^2(\Gamma_1)}, \\ M_{\partial,2}^{fg} &= \langle \phi_{\partial,2}^f, \phi_{\partial,2}^g \rangle_{L^2(\Gamma_2)}, \quad (f, g) \in \{1, n_{\partial,2}\}, & B_{1,\Gamma_2}^{ig} &= \langle \phi_1^i, \phi_{\partial,2}^g \rangle_{L^2(\Gamma_2)}, \end{aligned} \quad i \in \{1, n_1\}, \quad (7.113)$$

1299 the other matrices keep the same definition as in (7.54). The discrete Hamiltonian, whose
1300 expressive is [BMXZ18]

$$H_d = \frac{1}{2} \mathbf{e}_1^\top \mathbf{M}_{\mathcal{M}_1} \mathbf{e}_1 + \frac{1}{2} \mathbf{e}_2^\top \mathbf{M}_{\mathcal{M}_2} \mathbf{e}_2. \quad (7.114)$$

1301 gives rise to the discrete power balance

$$\begin{aligned} \dot{H}_d &= \mathbf{e}_1^\top \mathbf{M}_{\mathcal{M}_1} \dot{\mathbf{e}}_1 + \mathbf{e}_2^\top \mathbf{M}_{\mathcal{M}_2} \dot{\mathbf{e}}_2, \\ &= -\mathbf{e}_1^\top (\mathbf{D}_0 + \mathbf{D}_{\mathcal{L}})^\top \mathbf{e}_2 + \mathbf{e}_2^\top (\mathbf{D}_0 + \mathbf{D}_{\mathcal{L}}) \mathbf{e}_1 + \mathbf{e}_1^\top (\mathbf{B}_{1,\Gamma_1} \boldsymbol{\lambda}_{\partial,1} + \mathbf{B}_{1,\Gamma_2} \mathbf{u}_{\partial,2}), \\ &= \mathbf{y}_{\partial,1}^\top \mathbf{M}_{\partial,1} \mathbf{u}_{\partial,1} + \mathbf{y}_{\partial,2}^\top \mathbf{M}_{\partial,2} \mathbf{u}_{\partial,2}, \\ &= \hat{\mathbf{y}}_{\partial,1}^\top \mathbf{u}_{\partial,1} + \hat{\mathbf{y}}_{\partial,2}^\top \mathbf{u}_{\partial,2}, \quad \text{where} \quad \hat{\mathbf{y}}_{\partial,1} := \mathbf{M}_{\partial,1} \mathbf{y}_{\partial,1}, \quad \hat{\mathbf{y}}_{\partial,2} := \mathbf{M}_{\partial,2} \mathbf{y}_{\partial,2}. \end{aligned} \quad (7.115)$$

1302 This result is the finite dimensional equivalent of (7.106).

1303 Equivalently, the weak form Eq.7.53 may be used as a starting point. The computation
1304 follows in a completely analogous manner. The only difference is that $\mathbf{y}_{\partial,2} = \boldsymbol{\lambda}_{\partial,2}$ plays the
1305 role of the Lagrange multiplier. The final finite dimensional system then is given by

$$\begin{aligned} \text{Diag} \begin{bmatrix} \mathbf{M}_{\mathcal{M}_1} \\ \mathbf{M}_{\mathcal{M}_2} \\ \mathbf{0} \end{bmatrix} \begin{pmatrix} \dot{\mathbf{e}}_1 \\ \dot{\mathbf{e}}_2 \\ \dot{\boldsymbol{\lambda}}_{\partial,2} \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & -\mathbf{D}_0^\top + \mathbf{D}_{-\mathcal{L}^*} & \mathbf{0} \\ \mathbf{D}_0 - \mathbf{D}_{-\mathcal{L}^*}^\top & \mathbf{0} & \mathbf{B}_{2,\Gamma_2} \\ \mathbf{0} & -\mathbf{B}_{2,\Gamma_2}^\top & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \\ \boldsymbol{\lambda}_{\partial,2} \end{pmatrix} + \begin{bmatrix} \mathbf{0} & \mathbf{0} \\ \mathbf{B}_{2,\Gamma_1} & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_{\partial,2} \end{bmatrix} \begin{bmatrix} \mathbf{u}_{\partial,1} \\ \mathbf{u}_{\partial,2} \end{bmatrix}, \\ \begin{bmatrix} \mathbf{M}_{\partial,1} & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_{\partial,2} \end{bmatrix} \begin{pmatrix} \mathbf{y}_{\partial,1} \\ \mathbf{y}_{\partial,2} \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & \mathbf{B}_{2,\Gamma_1}^\top & \mathbf{0} \\ \mathbf{0} & \mathbf{0} & \mathbf{M}_{\partial,2} \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \\ \boldsymbol{\lambda}_{\partial,2} \end{pmatrix}. \end{aligned} \quad (7.116)$$

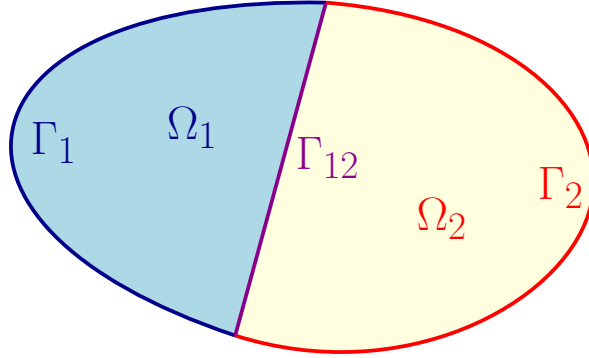
1306 where $\mathbf{B}_{2,\Gamma_1}, \mathbf{B}_{2,\Gamma_2}$ are given by

$$B_{2,\Gamma_1}^{mk} = \langle \phi_2^m, \phi_{\partial,1}^k \rangle_{L^2(\Gamma_1)}, \quad B_{2,\Gamma_2}^{mg} = \langle \phi_2^m, \phi_{\partial,2}^g \rangle_{L^2(\Gamma_2)}, \quad . \quad (7.117)$$

1307 where $m \in \{1, n_2\}, k \in \{1, n_{\partial,1}\}, g \in \{1, n_{\partial,2}\}$. This solution can be applied to incorporate
1308 all possible mixed boundary conditions in a systematic manner. However the finite element
1309 discretization is required to satisfy the inf-sup condition. Simulating the resulting system is
1310 harder, since the algebraic constraints pose additional difficulties for the time integration.

1311 7.2.2 Virtual domain decomposition

1312 Since the boundary are supposed to be connected set, a single interface is sufficient to obtain
1313 a system with the desired boundary conditions. In Fig. 7.2 the splitting of the domain is
1314 accomplished by introducing the interface Γ_{12} , that can be freely chosen. However, as a general

Figure 7.2: Splitting of the domain into Ω_1 , Ω_2 .

1315 guideline, it has to be such that the meshing of the subdomains generate regular triangles.

1316 The idea is based on the fact that System 7.104 can be split into two systems with uniform
 1317 causality. The following set of boundary variables is used for Ω_1 subdomain

$$\begin{pmatrix} \mathbf{u}_{\partial,1} \\ \mathbf{u}_{\partial,1}^{\Gamma_{12}} \end{pmatrix} = \begin{bmatrix} \mathcal{N}_{\partial,1}^{\Gamma_1} & 0 \\ \mathcal{N}_{\partial,1}^{\Gamma_{12}} & 0 \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix}, \quad \begin{pmatrix} \mathbf{y}_{\partial,1} \\ \mathbf{y}_{\partial,1}^{\Gamma_{12}} \end{pmatrix} = \begin{bmatrix} 0 & \mathcal{N}_{\partial,2}^{\Gamma_1} \\ 0 & \mathcal{N}_{\partial,2}^{\Gamma_{12}} \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix}. \quad (7.118)$$

1318 Whereas for the Ω_2 subdomain, the boundary variables are

$$\begin{pmatrix} \mathbf{u}_{\partial,2} \\ \mathbf{u}_{\partial,2}^{\Gamma_{12}} \end{pmatrix} = \begin{bmatrix} 0 & \mathcal{N}_{\partial,2}^{\Gamma_2} \\ 0 & \mathcal{N}_{\partial,2}^{\Gamma_{12}} \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix}, \quad \begin{pmatrix} \mathbf{y}_{\partial,2} \\ \mathbf{y}_{\partial,2}^{\Gamma_{12}} \end{pmatrix} = \begin{bmatrix} \mathcal{N}_{\partial,1}^{\Gamma_1} & 0 \\ \mathcal{N}_{\partial,1}^{\Gamma_{12}} & 0 \end{bmatrix} \begin{pmatrix} \mathbf{e}_1 \\ \mathbf{e}_2 \end{pmatrix}. \quad (7.119)$$

1319 The following relations then hold.

$$\mathbf{u}_{\partial,1}^{\Gamma_{12}} = \pm \mathbf{y}_{\partial,2}^{\Gamma_{12}}, \quad \mathbf{u}_{\partial,2}^{\Gamma_{12}} = \mp \mathbf{y}_{\partial,1}^{\Gamma_{12}}. \quad (7.120)$$

1320 The plus or minus sign is due to the fact that either $\mathcal{N}_{\partial,1}^{\Gamma_{12}}$ or $\mathcal{N}_{\partial,2}^{\Gamma_{12}}$ contains a scalar product
 1321 with the outgoing normal at Γ_{12} (that has opposite direction depending on which subdomain
 1322 is considered). These relations are at the core of the methodology, since they state the equiv-
 1323 alence between a problem with mixed causalities and the interconnection of two problems
 1324 with uniform causality.

1325

1326 To obtain a final system with the desired causality, the weak form has to be carried out
 1327 separately on each subdomain. In particular, on subdomain Ω_1 the \mathcal{L} operator is integrated
 1328 by parts, whereas on subdomain Ω_2 the $-\mathcal{L}^*$ operator undergoes the integration by parts.
 1329 Consequently, on subdomains Ω_1 (Ω_2) the boundary input $\mathbf{u}_{\partial,1}$ ($\mathbf{u}_{\partial,2}$) explicitly appears. Let
 1330 $X_1(\Omega_*)$ be the X_1 space restricted to the subdomain Ω_* , and let $X_2(\Omega_*)$ be the restriction
 1331 of X_2 to Ω_* for $* \in \{1, 2\}$. The weak form of the dynamics (7.104a) for the Ω_1 contribution
 1332 reads

$$\begin{aligned}
\langle \mathbf{v}_1, \mathcal{M}_1 \partial_t \mathbf{e}_1 \rangle_{X_1(\Omega_1)} &= -\langle \mathbf{v}_1, \mathbf{L}^\top \mathbf{e}_2 \rangle_{X_1(\Omega_1)} - \langle \mathbf{v}_1, \mathcal{L}^* \mathbf{e}_2 \rangle_{X_1(\Omega_1)} \\
\langle \mathbf{v}_2, \mathcal{M}_2 \partial_t \mathbf{e}_2 \rangle_{X_2(\Omega_1)} &= \langle \mathbf{v}_2, \mathbf{L} \mathbf{e}_1 \rangle_{X_2(\Omega_1)} + \langle \mathcal{L}^* \mathbf{v}_2, \mathbf{e}_1 \rangle_{X_1(\Omega_1)} + \langle \mathcal{N}_{\partial,2} \mathbf{v}_2, \mathcal{N}_{\partial,1} \mathbf{e}_1 \rangle_{L^2(\partial\Omega_1)}.
\end{aligned} \tag{7.121}$$

1333 For Ω_2 , we get

$$\begin{aligned}
\langle \mathbf{v}_1, \mathcal{M}_1 \partial_t \mathbf{e}_1 \rangle_{X_1(\Omega_2)} &= -\langle \mathbf{v}_1, \mathbf{L}^\top \mathbf{e}_2 \rangle_{X_1(\Omega_2)} - \langle \mathcal{L} \mathbf{v}_1, \mathbf{e}_2 \rangle_{X_2(\Omega_2)} + \langle \mathcal{N}_{\partial,1} \mathbf{v}_1, \mathcal{N}_{\partial,2} \mathbf{e}_2 \rangle_{L^2(\partial\Omega_2)}, \\
\langle \mathbf{v}_2, \mathcal{M}_2 \partial_t \mathbf{e}_2 \rangle_{X_2(\Omega_2)} &= \langle \mathbf{v}_2, \mathbf{L} \mathbf{e}_1 \rangle_{X_2(\Omega_2)} + \langle \mathbf{v}_2, \mathcal{L} \mathbf{e}_1 \rangle_{X_2(\Omega_2)},
\end{aligned} \tag{7.122}$$

1334 Since $\partial\Omega_1 = \bar{\Gamma}_1 \cup \bar{\Gamma}_{12}$ and $\partial\Omega_2 = \bar{\Gamma}_2 \cup \bar{\Gamma}_{12}$, the boundary terms can be decomposed

$$\begin{aligned}
\langle \mathcal{N}_{\partial,2} \mathbf{v}_2, \mathcal{N}_{\partial,1} \mathbf{e}_1 \rangle_{L^2(\partial\Omega_1)} &= \langle \mathcal{N}_{\partial,2} \mathbf{v}_2, \mathcal{N}_{\partial,1} \mathbf{e}_1 \rangle_{L^2(\Gamma_1)} + \langle \mathcal{N}_{\partial,2} \mathbf{v}_2, \mathcal{N}_{\partial,1} \mathbf{e}_1 \rangle_{L^2(\Gamma_{12})}, \\
&= \langle \mathcal{N}_{\partial,2}^{\Gamma_1} \mathbf{v}_2, \mathcal{N}_{\partial,1}^{\Gamma_1} \mathbf{e}_1 \rangle_{L^2(\Gamma_1)} + \langle \mathcal{N}_{\partial,2}^{\Gamma_{12}} \mathbf{v}_2, \mathcal{N}_{\partial,1}^{\Gamma_{12}} \mathbf{e}_1 \rangle_{L^2(\Gamma_{12})}, \\
&= \langle \mathcal{N}_{\partial,2}^{\Gamma_1} \mathbf{v}_2, \mathbf{u}_{\partial,1} \rangle_{L^2(\Gamma_1)} + \langle \mathcal{N}_{\partial,2}^{\Gamma_{12}} \mathbf{v}_2, \mathbf{u}_{\partial,1}^{\Gamma_{12}} \rangle_{L^2(\Gamma_{12})}.
\end{aligned} \tag{7.123}$$

1335 Analogously, for the remaining boundary term we find

$$\langle \mathcal{N}_{\partial,1} \mathbf{v}_1, \mathcal{N}_{\partial,2} \mathbf{e}_2 \rangle_{L^2(\partial\Omega_2)} = \langle \mathcal{N}_{\partial,1}^{\Gamma_2} \mathbf{v}_1, \mathbf{u}_{\partial,2} \rangle_{L^2(\Gamma_2)} + \langle \mathcal{N}_{\partial,1}^{\Gamma_{12}} \mathbf{v}_1, \mathbf{u}_{\partial,2}^{\Gamma_{12}} \rangle_{L^2(\Gamma_{12})}. \tag{7.124}$$

1336 A Galerkin approximation, analogous to (7.111), is used for each subdomain

$$\begin{aligned}
\mathbf{v}_{1,1} &\approx \sum_{i=1}^{n_{1,1}} \phi_{1,1}^i(\mathbf{x}_1) v_{1,1}^i, & \mathbf{x}_1 \in \Omega_1, & \quad \mathbf{v}_{1,2} \approx \sum_{i=1}^{n_{1,2}} \phi_{1,2}^i(\mathbf{x}_2) v_{1,2}^i, & \mathbf{x}_2 \in \Omega_2, \\
\mathbf{v}_{2,1} &\approx \sum_{i=1}^{n_{2,1}} \phi_{2,1}^i(\mathbf{x}_1) v_{2,1}^i, & & \quad \mathbf{v}_{2,2} \approx \sum_{i=1}^{n_{2,2}} \phi_{2,2}^i(\mathbf{x}_2) v_{2,2}^i, & \\
\mathbf{e}_{1,1} &\approx \sum_{i=1}^{n_{1,1}} \phi_{1,1}^i(\mathbf{x}_1) e_{1,1}^i(t), & & \quad \mathbf{e}_{1,2} \approx \sum_{i=1}^{n_{1,2}} \phi_{1,2}^i(\mathbf{x}_2) e_{1,2}^i(t), & \\
\mathbf{e}_{2,1} &\approx \sum_{i=1}^{n_{2,1}} \phi_{2,1}^i(\mathbf{x}_1) e_{2,1}^i(t), & & \quad \mathbf{e}_{2,2} \approx \sum_{i=1}^{n_{2,2}} \phi_{2,2}^i(\mathbf{x}_2) e_{2,2}^i(t). &
\end{aligned} \tag{7.125}$$

1337 For the boundary variables, additional terms for the common interface are needed

$$\begin{aligned}
\Box_{\partial,1} &\approx \sum_{i=1}^{n_{\partial,1}} \phi_{\partial,1}^i(\mathbf{s}_1) \Box_{\partial,1}^i(t), & \mathbf{s}_1 \in \Gamma_1, & \quad \Box_{\partial,1}^{\Gamma_{12}} \approx \sum_{i=1}^{n_{\partial,12}} \phi_{\partial,12}^i(\mathbf{s}_{12}) \Box_{\partial,1}^{i,\Gamma_{12}}(t), \\
& & & \quad \mathbf{s}_{12} \in \Gamma_{12}. \\
\Box_{\partial,2} &\approx \sum_{i=1}^{n_{\partial,2}} \phi_{\partial,2}^i(\mathbf{s}_2) \Box_{\partial,2}^i(t), & \mathbf{s}_2 \in \Gamma_2, & \quad \Box_{\partial,2}^{\Gamma_{12}} \approx \sum_{i=1}^{n_{\partial,12}} \phi_{\partial,12}^i(\mathbf{s}_{12}) \Box_{\partial,2}^{i,\Gamma_{12}}(t),
\end{aligned} \tag{7.126}$$

1338 where \Box stays for v, u, y . Notice that the same basis functions $\phi_{\partial,12}$ are used for both
1339 interface variables. This is necessary in order to dispose of the same degrees of freedom for
1340 the interconnection.

Replacing approximations 7.111, 7.126 into Eqs. 7.121, 7.123, 7.118, a finite dimensional system for the Ω_1 subdomain is obtained

$$\begin{aligned} \begin{bmatrix} \mathbf{M}_{\mathcal{M}_1}^{\Omega_1} & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_{\mathcal{M}_2}^{\Omega_1} \end{bmatrix} \begin{pmatrix} \dot{\mathbf{e}}_{1,1} \\ \dot{\mathbf{e}}_{2,1} \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & -\mathbf{D}_0^{\Omega_1 \top} + \mathbf{D}_{-\mathcal{L}^*}^{\Omega_1} \\ \mathbf{D}_0^{\Omega_1} - \mathbf{D}_{-\mathcal{L}^*}^{\Omega_1 \top} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_{1,1} \\ \mathbf{e}_{2,1} \end{pmatrix} + \begin{bmatrix} \mathbf{0} & \mathbf{0} \\ \mathbf{B}_{2,\Gamma_1}^{\Omega_1} & \mathbf{B}_{2,\Gamma_{12}}^{\Omega_1} \end{bmatrix} \begin{pmatrix} \mathbf{u}_{\partial,1} \\ \mathbf{u}_{\partial,1}^{\Gamma_{12}} \end{pmatrix}, \\ \begin{bmatrix} \mathbf{M}_{\partial,1} & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_{\partial,12} \end{bmatrix} \begin{pmatrix} \mathbf{y}_{\partial,1} \\ \mathbf{y}_{\partial,1}^{\Gamma_{12}} \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & \mathbf{B}_{2,\Gamma_1}^{\Omega_1 \top} \\ \mathbf{0} & \mathbf{B}_{2,\Gamma_{12}}^{\Omega_1 \top} \end{bmatrix} \begin{pmatrix} \mathbf{e}_{1,1} \\ \mathbf{e}_{2,1} \end{pmatrix}. \end{aligned} \quad (7.127)$$

The mass and interconnection operator matrices are the restriction to the subdomain of the matrices given in (7.116)

$$\begin{aligned} M_{\mathcal{M}_1}^{\Omega_1,ij} &= \langle \phi_{1,1}^i, \mathcal{M}_1 \phi_{1,1}^j \rangle_{X_1(\Omega_1)}, \quad D_0^{\Omega_1,mj} = \langle \phi_{2,1}^i, \mathbf{L} \phi_{1,1}^j \rangle_{X_2(\Omega_1)}, \quad i, j \in \{1, n_{1,1}\}, \\ M_{\mathcal{M}_2}^{\Omega_1,mn} &= \langle \phi_{2,1}^m, \mathcal{M}_2 \phi_{2,1}^n \rangle_{X_2(\Omega_1)}, \quad D_{-\mathcal{L}^*}^{\Omega_1,in} = \langle \phi_{1,1}^m, -\mathcal{L}^* \phi_{2,1}^n \rangle_{X_1(\Omega_1)}, \quad m, n \in \{1, n_{2,1}\}. \end{aligned} \quad (7.128)$$

Matrices $\mathbf{M}_{\partial,1}$ is constructed as in Eq. (7.116). Matrix $\mathbf{M}_{\partial,12}$ is similarly built

$$M_{\partial,12}^{lk} = \langle \phi_{\partial,12}^l, \phi_{\partial,12}^k \rangle_{L^2(\Gamma_{12})}, \quad l, k \in \{1, n_{\partial,12}\}. \quad (7.129)$$

The novel matrices $\mathbf{B}_{2,\Gamma_1}^{\Omega_1}$, $\mathbf{B}_{1,\Gamma_{12}}^{\Omega_1}$ have elements

$$\begin{aligned} B_{2,\Gamma_1}^{\Omega_1,mh} &= \langle \phi_{2,1}^m, \phi_{\partial,1}^h \rangle_{L^2(\Gamma_1)}, \quad m \in \{1, n_{2,1}\}, \quad h \in \{1, n_{\partial,1}\}, \\ B_{2,\Gamma_{12}}^{\Omega_1,mk} &= \langle \phi_{2,1}^m, \phi_{\partial,12}^k \rangle_{L^2(\Gamma_{12})}, \quad k \in \{1, n_{\partial,12}\}. \end{aligned} \quad (7.130)$$

If instead the approximations are plugged into Eqs. 7.122, 7.124, 7.119, a finite dimensional system for the Ω_2 subdomain is computed

$$\begin{aligned} \begin{bmatrix} \mathbf{M}_{\mathcal{M}_1}^{\Omega_2} & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_{\mathcal{M}_2}^{\Omega_2} \end{bmatrix} \begin{pmatrix} \dot{\mathbf{e}}_{1,2} \\ \dot{\mathbf{e}}_{2,2} \end{pmatrix} &= \begin{bmatrix} \mathbf{0} & -\mathbf{D}_0^{\Omega_2 \top} - \mathbf{D}_{\mathcal{L}}^{\Omega_2 \top} \\ \mathbf{D}_0^{\Omega_2} + \mathbf{D}_{\mathcal{L}}^{\Omega_2} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_{1,2} \\ \mathbf{e}_{2,2} \end{pmatrix} + \begin{bmatrix} \mathbf{B}_{1,\Gamma_2}^{\Omega_2} & \mathbf{B}_{1,\Gamma_{12}}^{\Omega_2} \\ \mathbf{0} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{u}_{\partial,2} \\ \mathbf{u}_{\partial,2}^{\Gamma_{12}} \end{pmatrix}, \\ \begin{bmatrix} \mathbf{M}_{\partial,2} & \mathbf{0} \\ \mathbf{0} & \mathbf{M}_{\partial,12} \end{bmatrix} \begin{pmatrix} \mathbf{y}_{\partial,2} \\ \mathbf{y}_{\partial,2}^{\Gamma_{12}} \end{pmatrix} &= \begin{bmatrix} \mathbf{B}_{1,\Gamma_2}^{\Omega_2 \top} & \mathbf{0} \\ \mathbf{B}_{1,\Gamma_{12}}^{\Omega_2 \top} & \mathbf{0} \end{bmatrix} \begin{pmatrix} \mathbf{e}_{1,2} \\ \mathbf{e}_{2,2} \end{pmatrix}. \end{aligned} \quad (7.131)$$

The mass and interconnection operator matrices are the restriction to the subdomain of the matrices given in (7.112)

$$\begin{aligned} M_{\mathcal{M}_1}^{\Omega_2,ij} &= \langle \phi_{1,2}^i, \mathcal{M}_1 \phi_{1,2}^j \rangle_{X_1(\Omega_2)}, \quad D_0^{\Omega_2,mj} = \langle \phi_{2,2}^i, \mathbf{L} \phi_{1,2}^j \rangle_{X_2(\Omega_2)}, \quad i, j \in \{1, n_{1,2}\}, \\ M_{\mathcal{M}_2}^{\Omega_2,mn} &= \langle \phi_{2,2}^m, \mathcal{M}_2 \phi_{2,2}^n \rangle_{X_2(\Omega_2)}, \quad D_{\mathcal{L}}^{\Omega_2,mj} = \langle \phi_{2,2}^m, \mathcal{L} \phi_{1,2}^j \rangle_{X_2(\Omega_2)}, \quad m, n \in \{1, n_{2,2}\}. \end{aligned} \quad (7.132)$$

Matrix $\mathbf{M}_{\partial,2}$ is constructed as in (7.112). The elements of matrices \mathbf{B}_{1,Γ_2} , $\mathbf{B}_{1,\Gamma_{12}}$ are computed as

$$\begin{aligned} B_{1,\Gamma_2}^{ig} &= \langle \phi_{1,2}^i, \phi_{\partial,2}^g \rangle_{L^2(\Gamma_2)}, & i \in \{1, n_{1,2}\}, & g \in \{1, n_{\partial,2}\}, \\ B_{1,\Gamma_{12}}^{ik} &= \langle \phi_{1,2}^i, \phi_{\partial,12}^k \rangle_{L^2(\Gamma_{12})}, & k \in \{1, n_{\partial,12}\}. \end{aligned} \quad (7.133)$$

Systems (7.127), (7.131) are compactly rewritten as

System (7.127)	System (7.131)
$\begin{aligned} \mathbf{M}_{\Omega_1} \dot{\mathbf{e}}_{\Omega_1} &= \mathbf{J}_{\Omega_1} \mathbf{e}_{\Omega_1} + \mathbf{B}_{\Gamma_1}^{\Omega_1} \mathbf{u}_{\partial,1} + \mathbf{B}_{\Gamma_{12}}^{\Omega_1} \mathbf{u}_{\partial,1}^{\Gamma_{12}}, \\ \mathbf{M}_{\partial,1} \mathbf{y}_{\partial,1} &= \mathbf{B}_{\Gamma_1}^{\Omega_1 \top} \mathbf{e}_{\Omega_1}, \\ \mathbf{M}_{\partial,12} \mathbf{y}_{\partial,1}^{\Gamma_{12}} &= \mathbf{B}_{\Gamma_{12}}^{\Omega_1 \top} \mathbf{e}_{\Omega_1}. \end{aligned} \quad (7.134)$	$\begin{aligned} \mathbf{M}_{\Omega_2} \dot{\mathbf{e}}_{\Omega_2} &= \mathbf{J}_{\Omega_2} \mathbf{e}_{\Omega_2} + \mathbf{B}_{\Gamma_2}^{\Omega_2} \mathbf{u}_{\partial,2} + \mathbf{B}_{\Gamma_{12}}^{\Omega_2} \mathbf{u}_{\partial,2}^{\Gamma_{12}}, \\ \mathbf{M}_{\partial,2} \mathbf{y}_{\partial,2} &= \mathbf{B}_{\Gamma_2}^{\Omega_2 \top} \mathbf{e}_{\Omega_2}, \\ \mathbf{M}_{\partial,12} \mathbf{y}_{\partial,2}^{\Gamma_{12}} &= \mathbf{B}_{\Gamma_{12}}^{\Omega_2 \top} \mathbf{e}_{\Omega_2}. \end{aligned} \quad (7.135)$

To obtain a system with the desired causality, an interconnection is employed to connect the two Systems (7.134), (7.135) along the shared boundary Γ_{12} . Given (7.120), the interconnection is computed as

$$\begin{aligned} \mathbf{u}_{\partial,1}^{\Gamma_{12}} &= \pm \mathbf{y}_{\partial,2}^{\Gamma_{12}} = \pm \mathbf{M}_{\partial,12}^{-1} \mathbf{B}_{\Gamma_{12}}^{\Omega_2 \top} \mathbf{e}_{\Omega_2}, \\ \mathbf{u}_{\partial,2}^{\Gamma_{12}} &= \mp \mathbf{y}_{\partial,1}^{\Gamma_{12}} = \mp \mathbf{M}_{\partial,12}^{-1} \mathbf{B}_{\Gamma_{12}}^{\Omega_1 \top} \mathbf{e}_{\Omega_1}, \end{aligned} \quad (7.136)$$

This corresponds to a classical gyrator interconnection of pH system.

This solution requires suitable finite elements for both kind of discretization detailed in Sec. 7.1.1. Some particular cases of boundary control cannot be handled nevertheless. The interface that separates the domain is an additional degree of freedom, as it can be freely drawn.

7.3 Conclusion

Convergence numerical study

1367 **8.1** Plate problems using known mixed finite elements

1368 **8.2** Non-standard discretization of flexible structures

Numerical applications

1372 9.1 Boundary stabilization

1373 9.2 Thermoelastic wave propagation

1374 9.3 Mixed boundary conditions

1375 9.3.1 Trajectory tracking of a thin beam

1376 9.3.2 Vibroacoustic under mixed boundary conditions

1377 9.4 Modal analysis of plates

1378

Part IV

1379

Port-Hamiltonian flexible multibody dynamics

1380

Modular multibody systems in port-Hamiltonian form

10.1 Reminder of the rigid case

10.2 Flexible floating body

10.3 Modular construction of multibody systems

1388

CHAPTER 11

1389

Validation

1390

1391 11.1 Beam systems

1392 11.1.1 Modal analysis of a flexible mechanism

1393 11.1.2 Non-linear crank slider

1394 11.1.3 Hinged beam

1395 11.2 Plate systems

1396 11.2.1 Boundary interconnection with a rigid element

1397 11.2.2 Actuated plate

Conclusion

Conclusions and future directions

Je n'ai cherché de rien prouver, mais de bien peindre et d'éclairer bien ma
peinture.

André Gide
Préface de L'Immoraliste

Mathematical tools

A.1 Differential operators

The space of all, symmetric and skew-symmetric $d \times d$ matrices are denoted by \mathbb{M} , \mathbb{S} , \mathbb{K} respectively. The space of \mathbb{R}^d vectors is denoted by \mathbb{V} . $\Omega \subset \mathbb{R}^d$ is an open connected set. For a scalar field $u : \Omega \rightarrow \mathbb{R}$ the gradient is defined as

$$\text{grad}(u) = \nabla u := \left(\partial_{x_1} u \dots \partial_{x_d} u \right)^\top.$$

For a vector field $\mathbf{u} : \Omega \rightarrow \mathbb{V}$, with components u_i , the gradient (Jacobian) is defined as

$$\text{grad}(\mathbf{u})_{ij} := (\nabla \mathbf{u})_{ij} = \partial_{x_i} u_j.$$

The symmetric part of the gradient operator Grad (i. e. the deformation gradient in continuum mechanics) is thus given by

$$\text{Grad}(\mathbf{u}) := \frac{1}{2} \left(\nabla \mathbf{u} + (\nabla \mathbf{u})^\top \right) \in \mathbb{S}.$$

The Hessian operator of u is then computed as follows

$$\text{Hess}(u) = \nabla^2 u = \text{Grad}(\text{grad}(u)),$$

For a tensor field $\mathbf{U} : \Omega \rightarrow \mathbb{M}$, with components u_{ij} , the divergence is a vector, defined column-wise as

$$\text{Div}(\mathbf{U}) = \nabla \cdot \mathbf{U} := \left(\sum_{i=1}^d \partial_{x_i} u_{ij} \right)_{j=1, \dots, d}.$$

The double divergence of a tensor field \mathbf{U} is then a scalar field defined as

$$\text{div}(\text{Div}(\mathbf{U})) := \sum_{i=1}^d \sum_{j=1}^d \partial_{x_i} \partial_{x_j} u_{ij}.$$

Definition 6 (Formal adjoint, Def. 5.80 [RR04])

Consider the differential operator defined on Ω

$$\mathcal{L}(\mathbf{x}, \partial) = \sum_{|\alpha| \leq k} a_\alpha(\mathbf{x}) \partial^\alpha, \tag{A.1}$$

where $\alpha := (\alpha_1, \dots, \alpha_d)$ is a multi-index of order $|\alpha| := \sum_{i=1}^d \alpha_i$, a_α are a set of real scalars and $\partial^\alpha := \partial_{x_1}^{\alpha_1} \dots \partial_{x_d}^{\alpha_d}$ is a differential operator of order $|\alpha|$ resulting from a combination of spatial derivatives. The formal adjoint of \mathcal{L} is the operator defined by

$$\mathcal{L}^*(\mathbf{x}, \partial)u = \sum_{|\alpha| \leq k} (-1)^\alpha \partial^\alpha (a_\alpha(\mathbf{x})u(\mathbf{x})). \quad (\text{A.2})$$

The importance of this definition lies in the fact that

$$\langle \phi, \mathcal{L}(\mathbf{x}, \partial)\psi \rangle_\Omega = \langle \mathcal{L}^*(\mathbf{x}, \partial)\phi, \psi \rangle_\Omega \quad (\text{A.3})$$

for every $\phi, \psi \in C_0^\infty(\Omega)$. If the assumption of compact support is removed, then (A.3) no longer holds; instead the integration by parts yields additional terms involving integrals over the boundary $\partial\Omega$. However, these boundary terms vanish if ϕ and ψ satisfy certain restrictions on the boundary.

A.2 Integration by parts

Theorem 5 (Integration by parts for tensors)

Consider a smooth tensor-valued function $\mathbf{A} \in \mathbb{R}^{d \times d}$ and vector-valued function $\mathbf{b} \in \mathbb{V} = \mathbb{R}^d$. The following integration by parts formula holds

$$\int_\Omega \{\text{Div}(\mathbf{A}) \cdot \mathbf{b} + \mathbf{A} : \text{grad}(\mathbf{b})\} \, d\Omega = \int_\Omega \text{div}(\mathbf{A}\mathbf{b}) \, d\Omega = \int_{\partial\Omega} (\mathbf{A}^\top \mathbf{n}) \cdot \mathbf{b} \, dS, \quad (\text{A.4})$$

where \mathbf{n} is the outward normal at the boundary and dS the infinitesimal surface.

Proof. Consider the components expression of Eq. (A.4)

$$\begin{aligned} \int_\Omega \{\text{Div}(\mathbf{A}) \cdot \mathbf{b} + \mathbf{A} : \text{grad}(\mathbf{b})\} \, d\Omega &= \int_\Omega \sum_{i=1}^d \sum_{j=1}^d \{(\partial_{x_i} A_{ij})b_j + A_{ij}(\partial_{x_i} b_j)\} \, d\Omega, \\ &= \int_\Omega \sum_{i=1}^d \sum_{j=1}^d \partial_{x_i} (A_{ij}b_j) \, d\Omega = \int_\Omega \text{div}(\mathbf{A}\mathbf{b}) \, d\Omega, \\ &= \int_{\partial\Omega} \sum_{i=1}^d \sum_{j=1}^d (n_i A_{ij})b_j \, dS = \int_{\partial\Omega} (\mathbf{A}^\top \mathbf{n}) \cdot \mathbf{b} \, dS. \end{aligned} \quad (\text{A.5})$$

□

The previous result can be specialized for symmetric tensor field [BBF⁺13, Chapter 1].

Theorem 6 (Integration by parts for symmetric tensors)

Consider a smooth tensor-valued function $\mathbf{M} \in \mathbb{S} = \mathbb{R}_{sym}^{d \times d}$ and vector-valued function $\mathbf{b} \in \mathbb{V} =$

1425 \mathbb{R}^d . Then, it holds

$$\int_{\Omega} \{\text{Div}(\mathbf{S}) \cdot \mathbf{b} + \mathbf{M} : \text{Grad}(\mathbf{b})\} \, d\Omega = \int_{\Omega} \text{div}(\mathbf{M}\mathbf{b}) \, d\Omega = \int_{\partial\Omega} (\mathbf{M}\mathbf{n}) \cdot \mathbf{b} \, dS. \quad (\text{A.6})$$

1426 *Proof.* Consider the components expression of Eq. (A.6)

$$\int_{\Omega} \{\text{Div}(\mathbf{M}) \cdot \mathbf{b} + \mathbf{M} : \text{Grad}(\mathbf{b})\} \, d\Omega = \int_{\Omega} \sum_{i=1}^d \sum_{j=1}^d \left\{ (\partial_{x_i} M_{ij}) b_j + M_{ij} \frac{1}{2} (\partial_{x_i} b_j + \partial_{x_j} b_i) \right\} \, d\Omega, \quad (\text{A.7})$$

1427 The term $M_{ij} \frac{1}{2} (\partial_{x_i} b_j + \partial_{x_j} b_i)$ can be manipulated exploiting the symmetry of the tensor \mathbf{M}

$$\begin{aligned} \sum_{i=1}^d \sum_{j=1}^d \frac{1}{2} (M_{ij} \partial_{x_i} b_j + M_{ij} \partial_{x_j} b_i) &= \sum_{i=1}^d \sum_{j=1}^d \frac{1}{2} (M_{ij} \partial_{x_i} b_j + M_{ji} \partial_{x_i} b_j), \\ &= \sum_{i=1}^d \sum_{j=1}^d \frac{1}{2} (M_{ij} + M_{ji}) \partial_{x_i} b_j \quad \text{Since } \mathbf{M} \text{ is symmetric,} \\ &= \sum_{i=1}^d \sum_{j=1}^d M_{ij} \partial_{x_i} b_j = \mathbf{M} : \text{grad}(\mathbf{b}) \end{aligned} \quad (\text{A.8})$$

1428 Then it holds

$$\int_{\Omega} \{\text{Div}(\mathbf{M}) \cdot \mathbf{b} + \mathbf{M} : \text{Grad}(\mathbf{b})\} \, d\Omega = \int_{\Omega} \{\text{Div}(\mathbf{M}) \cdot \mathbf{b} + \mathbf{M} : \text{grad}(\mathbf{b})\} \, d\Omega \quad (\text{A.9})$$

1429 Using Eq (A.4) then

$$\begin{aligned} \int_{\Omega} \{\text{Div}(\mathbf{M}) \cdot \mathbf{b} + \mathbf{M} : \text{Grad}(\mathbf{b})\} \, d\Omega &= \int_{\Omega} \{\text{Div}(\mathbf{M}) \cdot \mathbf{b} + \mathbf{M} : \text{grad}(\mathbf{b})\} \, d\Omega, \\ &= \int_{\partial\Omega} (\mathbf{M}^{\top} \mathbf{n}) \cdot \mathbf{b} \, dS, \quad \text{Since } \mathbf{M} \text{ is symmetric,} \\ &= \int_{\partial\Omega} (\mathbf{M} \mathbf{n}) \cdot \mathbf{b} \, dS. \end{aligned} \quad (\text{A.10})$$

1430 This concludes the proof. \square

1431 A.3 Bilinear forms

Definition 7 (Skew-symmetric bilinear form)

A bilinear form on the Hilbert space H

$$\begin{aligned} b : H \times H &\longrightarrow \mathbb{R}, \\ (\mathbf{v}, \mathbf{u}) &\longrightarrow b(\mathbf{v}, \mathbf{u}), \end{aligned}$$

is skew-symmetric iff

$$b(\boldsymbol{v}, \boldsymbol{u}) = -b(\boldsymbol{u}, \boldsymbol{v}).$$

Finite elements gallery

1435

APPENDIX C

1436

Implementation using FEniCS and Firedrake

1437

1438

Bibliography

1439

- 1440 [AB85] Douglas N Arnold and Franco Brezzi. Mixed and nonconforming finite ele-
1441 ment methods: implementation, postprocessing and error estimates. *ESAIM:*
1442 *Mathematical Modelling and Numerical Analysis-Modélisation Mathématique et*
1443 *Analyse Numérique*, 19(1):7–32, 1985.
- 1444 [Abe12] R. Abeyaratne. *Lecture Notes on the Mechanics of Elastic Solids. Volume II:*
1445 *Continuum Mechanics*. Cambridge, MA and Singapore, 1st edition, 2012.
- 1446 [AFS68] J. H. Argyris, I. Fried, and D. W. Scharpf. The tuba family of plate ele-
1447 ments for the matrix displacement method. *The Aeronautical Journal (1968)*,
1448 72(692):701–709, 1968.
- 1449 [AFW07] D. Arnold, R. Falk, and R. Winther. Mixed finite element methods for lin-
1450 ear elasticity with weakly imposed symmetry. *Mathematics of Computation*,
1451 76(260):1699–1723, 2007.
- 1452 [AL00] G. Avalos and I. Lasiecka. Boundary controllability of thermoelastic plates via
1453 the free boundary conditions. *SIAM Journal on Control and Optimization*,
1454 38(2):337–383, 2000.
- 1455 [AW02] D. Arnold and R. Winther. Mixed finite elements for elasticity. *Numerische*
1456 *Mathematik*, 92(3):401–419, 2002.
- 1457 [BBF⁺13] D. Boffi, F. Brezzi, M. Fortin, et al. *Mixed finite element methods and applica-*
1458 *tions*, volume 44. Springer, 2013.
- 1459 [Bel69] K. Bell. A refined triangular plate bending finite element. *International Journal*
1460 *for Numerical Methods in Engineering*, 1(1):101–122, 1969.
- 1461 [Bio56] M. A. Biot. Thermoelasticity and irreversible thermodynamics. *Journal of*
1462 *Applied Physics*, 27(3):240–253, 1956.
- 1463 [BMXZ18] C. Beattie, V. Mehrmann, H. Xu, and H. Zwart. Linear port-Hamiltonian de-
1464 scriptor systems. *Mathematics of Control, Signals, and Systems*, 30(4):17, 2018.
- 1465 [BR90] H. Blum and R. Rannacher. On mixed finite element methods in plate bending
1466 analysis. *Computational Mechanics*, 6(3):221–236, May 1990.
- 1467 [Bre08] F. Brezzi. *Mixed finite elements, compatibility conditions, and applications*.
1468 Springer, 2008.
- 1469 [Car73] D. E. Carlson. Linear thermoelasticity. In C. Truesdell, editor, *Linear Theo-*
1470 *ries of Elasticity and Thermoelasticity: Linear and Nonlinear Theories of Rods,*
1471 *Plates, and Shells*, pages 297–345. Springer, Berlin, Heidelberg, 1973.

-
- 1472 [Cha62] P Chadwick. On the propagation of thermoelastic disturbances in thin plates
1473 and rods. *Journal of the Mechanics and Physics of Solids*, 10(2):99–109, 1962.
- 1474 [Cia88] P. G. Ciarlet. *Mathematical Elasticity: Three-Dimensional Elasticity*. Studies
1475 in mathematics and its applications. North-Holland, 1988.
- 1476 [CMKO11] S. H. Christiansen, H. Z. Munthe-Kaas, and B. Owren. Topics in structure-
1477 preserving discretization. *Acta Numerica*, 20:1–119, 2011.
- 1478 [Cou90] T.J. Courant. Dirac manifolds. *Transactions of the American Mathematical
1479 Society*, 319(2):631–661, 1990.
- 1480 [CR16] F.L. Cardoso Ribeiro. *Port-Hamiltonian modeling and control of fluid-structure
1481 system*. PhD thesis, Université de Toulouse, Dec. 2016.
- 1482 [CRML18] F.L. Cardoso-Ribeiro, D. Matignon, and L. Lefèvre. A structure-preserving par-
1483 titioned finite element method for the 2d wave equation. *IFAC-PapersOnLine*,
1484 51(3):119 – 124, 2018. 6th IFAC Workshop on Lagrangian and Hamiltonian
1485 Methods for Nonlinear Control LHMNC 2018.
- 1486 [CRML19] F. L. Cardoso-Ribeiro, D. Matignon, and L. Lefèvre. A partitioned finite element
1487 method for power-preserving discretization of open systems of conservation laws,
1488 2019. arXiv preprint arXiv:1906.05965.
- 1489 [CRMPB17] F. L. Cardoso-Ribeiro, D. Matignon, and V. Pommier-Budinger. A port-
1490 Hamiltonian model of liquid sloshing in moving containers and application to a
1491 fluid-structure system. *Journal of Fluids and Structures*, 69:402–427, February
1492 2017.
- 1493 [DHNLS99] R. Durán, L. Hervella-Nieto, E. Liberman, and J. Solomin. Approximation of
1494 the vibration modes of a plate by Reissner-Mindlin equations. *Mathematics of
1495 Computation of the American Mathematical Society*, 68(228):1447–1463, 1999.
- 1496 [DMSB09] V. Duindam, A. Macchelli, S. Stramigioli, and H. Bruyninckx. *Modeling and
1497 Control of Complex Physical Systems*. Springer Verlag, 2009.
- 1498 [DZ18] Pauly D. and W. Zulehner. The divdiv-complex and applications to biharmonic
1499 equations. *Applicable Analysis*, pages 1–52, 2018.
- 1500 [Gri15] M. Grinfeld. *Mathematical Tools for Physicists*. John Wiley & Sons Inc, 2nd
1501 edition, jan 2015.
- 1502 [GSV18] T. Gustafsson, R. Stenberg, and J. Videman. A posteriori estimates for con-
1503 forming kirchhoff plate elements. *SIAM Journal on Scientific Computing*,
1504 40(3):A1386–A1407, 2018.
- 1505 [HE09] R. B. Hetnarski and M. R. Eslami. *Thermal stresses: advanced theory and
1506 applications*, volume 158. Springer, 2009.
-

-
- 1507 [HM78] T. J.R. Hughes and J.E. Marsden. Classical elastodynamics as a linear symmet-
1508 ric hyperbolic system. *Journal of Elasticity*, 8(1):97–110, 1978.
- 1509 [JZ12] B. Jacob and H. Zwart. *Linear Port-Hamiltonian Systems on Infinite-*
1510 *dimensional Spaces*. Number 223 in Operator Theory: Advances and Ap-
1511 plications. Springer Verlag, Germany, 2012. [https://doi.org/10.1007/](https://doi.org/10.1007/978-3-0348-0399-1)
1512 [978-3-0348-0399-1](https://doi.org/10.1007/978-3-0348-0399-1).
- 1513 [KML18] P. Kotyczka, B. Maschke, and L. Lefèvre. Weak form of Stokes-Dirac structures
1514 and geometric discretization of port-Hamiltonian systems. *Journal of Compu-*
1515 *tational Physics*, 361:442 – 476, 2018.
- 1516 [Kot19] P. Kotyczka. *Numerical Methods for Distributed Parameter Port-Hamiltonian*
1517 *Systems*. TUM University Press, 2019.
- 1518 [KZ15] M. Kurula and H. Zwart. Linear wave systems on n-d spatial domains. *Interna-*
1519 *tional Journal of Control*, 88(5):1063–1077, 2015. [https://www.tandfonline.](https://www.tandfonline.com/doi/abs/10.1080/00207179.2014.993337)
1520 [com/doi/abs/10.1080/00207179.2014.993337](https://www.tandfonline.com/doi/abs/10.1080/00207179.2014.993337).
- 1521 [KZvdSB10] M. Kurula, H. Zwart, A. J. van der Schaft, and J. Behrndt. Dirac structures
1522 and their composition on Hilbert spaces. *Journal of mathematical analysis and*
1523 *applications*, 372(2):402–422, 2010. [https://doi.org/10.1016/j.jmaa.2010.](https://doi.org/10.1016/j.jmaa.2010.07.004)
1524 [07.004](https://doi.org/10.1016/j.jmaa.2010.07.004).
- 1525 [Lag89] J. E. Lagnese. *Boundary Stabilization of Thin Plates*. Society for Industrial and
1526 Applied Mathematics, 1989.
- 1527 [Lee12] J. Lee. *Mixed methods with weak symmetry for time dependent problems of*
1528 *elasticity and viscoelasticity*. PhD thesis, University of Minnesota, 2012.
- 1529 [LGZM05] Y. Le Gorrec, H. Zwart, and B. Maschke. Dirac structures and Boundary Control
1530 Systems associated with Skew-Symmetric Differential Operators. *SIAM Journal*
1531 *on Control and Optimization*, 44(5):1864–1892, 2005. [https://doi.org/10.](https://doi.org/10.1137/040611677)
1532 [1137/040611677](https://doi.org/10.1137/040611677).
- 1533 [LMW⁺12] A. Logg, K. A. Mardal, G. N. Wells, et al. *Automated Solution of Differential*
1534 *Equations by the Finite Element Method*. Springer, 2012.
- 1535 [LPKL12] L. D. Landau, L. P. Pitaevskii, A. M. Kosevich, and E. M. Lifshitz. *Theory of*
1536 *Elasticity*. Butterworth Heinemann, third edition, Dec 2012.
- 1537 [LR00] R. Lifshitz and M. L. Roukes. Thermoelastic damping in micro-and nanome-
1538 chanical systems. *Physical review B*, 61(8):5600, 2000.
- 1539 [Min51] R. D. Mindlin. Influence of rotatory inertia and shear on flexural motions of
1540 isotropic elastic Plates. *Journal of Applied Mechanics*, 18:31–38, March 1951.
- 1541 [MM19] V. Mehrmann and R. Morandin. Structure-preserving discretization for port-
1542 hamiltonian descriptor systems. In *2019 IEEE 58th Conference on Decision and*
1543 *Control (CDC)*, pages 6863–6868, 2019.
-

-
- 1544 [MMB05] A. Macchelli, C. Melchiorri, and L. Bassi. Port-based modelling and control of
1545 the Mindlin plate. In *Proceedings of the 44th IEEE Conference on Decision and*
1546 *Control*, pages 5989–5994, Dec. 2005. [https://doi.org/10.1109/CDC.2005.](https://doi.org/10.1109/CDC.2005.1583120)
1547 [1583120](https://doi.org/10.1109/CDC.2005.1583120).
- 1548 [MvdSM04] A. Macchelli, A. J. van der Schaft, and C. Melchiorri. Port Hamiltonian formu-
1549 lation of infinite dimensional systems I. Modeling. In *Proceedings of the 43th*
1550 *IEEE Conference on Decision and Control*, volume 4, pages 3762–3767. IEEE,
1551 Dec. 2004.
- 1552 [Nor06] A.N. Norris. Dynamics of thermoelastic thin plates: A comparison of four
1553 theories. *Journal of Thermal Stresses*, 29(2):169–195, 2006.
- 1554 [Olv93] P. J. Olver. *Applications of Lie groups to differential equations*, volume 107 of
1555 *Graduate texts in mathematics*. Springer-Verlag New York, 2nd edition, 1993.
- 1556 [Pir89] O. A. Pironneau. *Finite element methods for fluids*. John Wiley and Sons, 1989.
- 1557 [PZ20] D. Pauly and W. Zulehner. The elasticity complex, 2020. arXiv preprint
1558 arXiv:2001.11007.
- 1559 [Red03] J. N. Reddy. *Mechanics of laminated composite plates and shells: theory and*
1560 *analysis*. CRC press, 2003.
- 1561 [Red06] J. N. Reddy. *Theory and analysis of elastic plates and shells*. CRC press, 2006.
- 1562 [Rei47] E. Reissner. On bending of elastic plates. *Quarterly of Applied Mathematics*,
1563 5(1):55–68, 1947.
- 1564 [RHM⁺17] F. Rathgeber, D.A. Ham, L. Mitchell, M. Lange, F. Luporini, A. T.T. McRae,
1565 G.T. Bercea, G. R. Markall, and P.H.J. Kelly. Firedrake: automating the finite
1566 element method by composing abstractions. *ACM Transactions on Mathemat-*
1567 *ical Software (TOMS)*, 43(3):24, 2017.
- 1568 [RR04] M. Renardy and R. C. Rogers. *An Introduction to Partial Differential Equations*.
1569 Number 13 in Texts in Applied Mathematics. Springer-Verlag New York, 2nd
1570 edition, 2004.
- 1571 [RZ18] K. Rafetseder and W. Zulehner. A decomposition result for Kirchhoff plate bend-
1572 ing problems and a new discretization approach. *SIAM Journal on Numerical*
1573 *Analysis*, 56(3):1961–1986, 2018.
- 1574 [SHM19a] A. Serhani, G. Haine, and D. Matignon. Anisotropic heterogeneous n-D
1575 heat equation with boundary control and observation: I. Modeling as port-
1576 Hamiltonian system. *IFAC-PapersOnLine*, 52(7):51 – 56, 2019. 3rd IFAC
1577 Workshop on Thermodynamic Foundations for a Mathematical Systems The-
1578 ory TFMST 2019.
-

-
- 1579 [SHM19b] A. Serhani, G. Haine, and D. Matignon. Anisotropic heterogeneous n-D heat
 1580 equation with boundary control and observation: II. Structure-preserving dis-
 1581 cretization. *IFAC-PapersOnLine*, 52(7):57 – 62, 2019. 3rd IFAC Workshop
 1582 on Thermodynamic Foundations for a Mathematical Systems Theory TFMST
 1583 2019.
- 1584 [Sim99] J. G. Simmonds. Major simplifications in a current linear model for the motion
 1585 of a thermoelastic plate. *Quarterly of Applied Mathematics*, 57(4):673–679, 1999.
- 1586 [Skr19] N. Skrepek. Well-posedness of linear first order port-Hamiltonian systems on
 1587 multidimensional spatial domains, 2019. arXiv preprint arXiv:1910.09847.
- 1588 [SS17] M. Schöberl and K. Schlacher. Variational Principles for Different Represen-
 1589 tations of Lagrangian and Hamiltonian Systems. In Hans Irschik, Alexander
 1590 Belyaev, and Michael Krommer, editors, *Dynamics and Control of Advanced*
 1591 *Structures and Machines*, pages 65–73. Springer International Publishing, 2017.
- 1592 [TRLGK18] V. Trenchant, H. Ramírez, Y. Le Gorrec, and P. Kotyczka. Finite differences
 1593 on staggered grids preserving the port-Hamiltonian structure with application
 1594 to an acoustic duct. *Journal of Computational Physics*, 373, 06 2018.
- 1595 [TW09] M. Tucsnak and G. Weiss. *Observation and control for operator semigroups*.
 1596 Springer Science & Business Media, 2009.
- 1597 [TWK59] S. Timoshenko and S. Woinowsky-Krieger. *Theory of plates and shells*. Engi-
 1598 neering societies monographs. McGraw-Hill, 1959.
- 1599 [vdSM02] A.J. van der Schaft and B. Maschke. Hamiltonian formulation of distributed-
 1600 parameter systems with boundary energy flow. *Journal of Geometry and*
 1601 *Physics*, 42(1):166 – 194, 2002.
- 1602 [Vil07] J.A. Villegas. *A Port-Hamiltonian Approach to Distributed Parameter Systems*.
 1603 PhD thesis, University of Twente, May 2007.
- 1604 [Yao11] P.F. Yao. *Modeling and Control in Vibrational and Structural Dynamics: A*
 1605 *Differential Geometric Approach*. Chapman & Hall/CRC Applied Mathematics
 1606 & Nonlinear Science. Taylor & Francis, 2011.
-

Résumé — Malgré l’abondante littérature sur le formalisme pH, les problèmes d’élasticité en deux ou trois dimensions géométriques n’ont presque jamais été considérés. Cette thèse vise à étendre l’approche port-Hamiltonienne (pH) à la mécanique des milieux continus. L’originalité apportée réside dans trois contributions majeures. Tout d’abord, la nouvelle formulation pH des modèles de plaques et des phénomènes thermoélastiques couplés est présentée. L’utilisation du calcul tensoriel est obligatoire pour modéliser les milieux continus et l’introduction de variables tensorielles est nécessaire pour obtenir une description pH équivalente qui soit intrinsèque, c’est-à-dire indépendante des coordonnées choisies. Deuxièmement, une technique de discrétisation basée sur les éléments finis et capable de préserver la structure du problème de la dimension infinie au niveau discret est développée et validée. La discrétisation des problèmes d’élasticité nécessite l’utilisation d’éléments finis non standard. Néanmoins, l’implémentation numérique est réalisée grâce à des bibliothèques open source bien établies, fournissant aux utilisateurs externes un outil facile à utiliser pour simuler des systèmes flexibles sous forme pH. Troisièmement, une nouvelle formulation pH de la dynamique multicorps flexible est dérivée. Cette reformulation, valable sous de petites hypothèses de déformations, inclut toutes sortes de modèles élastiques linéaires et exploite la modularité intrinsèque des systèmes pH.

Mots clés : Systèmes port-Hamiltonien, mécanique des solides, discretisation symplectique, méthode des éléments finis, dynamique multicorps

Abstract — Despite the large literature on pH formalism, elasticity problems in higher geometrical dimensions have almost never been considered. This work establishes the connection between port-Hamiltonian distributed systems and elasticity problems. The originality resides in three major contributions. First, the novel pH formulation of plate models and coupled thermoelastic phenomena is presented. The use of tensor calculus is mandatory for continuum mechanical models and the inclusion of tensor variables is necessary to obtain an equivalent and intrinsic, i.e. coordinate free, pH description. Second, a finite element based discretization technique, capable of preserving the structure of the infinite-dimensional problem at a discrete level, is developed and validated. The discretization of elasticity problems requires the use of non-standard finite elements. Nevertheless, the numerical implementation is performed thanks to well-established open-source libraries, providing external users with an easy to use tool for simulating flexible systems in pH form. Third, flexible multibody systems are recast in pH form by making use of a floating frame description valid under small deformations assumptions. This reformulation include all kinds of linear elastic models and exploits the intrinsic modularity of pH systems.

Keywords: Port-Hamiltonian systems, continuum mechanics, structure preserving discretization, finite element method, multibody dynamics.
