

Partitioned Finite Element Method for the Mindlin Plate as a Port-Hamiltonian system

Andrea Brugnoli

May 7, 2019

- 1 PH formulation of the Mindlin plate
 - Mindlin-Reissner model for thick Plates
 - Port-Hamiltonian formulation
- 2 Structure preserving discretization
 - Boundary control through forces and momenta
 - Boundary control through kinematic variables
- 3 Discretization procedure
 - Finite-dimensional system
- 4 Numerical simulations

- 1 PH formulation of the Mindlin plate
 - Mindlin-Reissner model for thick Plates
 - Port-Hamiltonian formulation
- 2 Structure preserving discretization
- 3 Discretization procedure
- 4 Numerical simulations

The Mindlin-Reissner model

The classical model is a system 3PDEs:

$$\begin{cases} \rho h \frac{\partial^2 w}{\partial t^2} &= \operatorname{div}(\mathbf{q}), \\ \rho \frac{h^3}{12} \frac{\partial^2 \boldsymbol{\theta}}{\partial t^2} &= \mathbf{q} + \operatorname{Div}(\mathbf{M}), \end{cases}$$

with the parameters and variables

- ρ the material density;
- h the plate thickness;
- the vertical displacement scalar field w ;
- the cross section deflection vector field $\boldsymbol{\theta} = (\theta_x, \theta_y)$;
- the bending symmetric tensor field \mathbf{M} ;
- shear stress vector field \mathbf{q} ;

The divergence of a tensor field is a vector defined column-wise as

$$\operatorname{Div}(\mathbf{M}) := \left(\sum_{\alpha=1}^2 \partial_{x_\alpha} m_{\alpha\beta} \right)_{\beta=1,\dots,2}.$$

Constitutive equations

For an homogeneous, isotropic material (Greek indexes equal 1,2)

$$\mathbf{M}_{\alpha\beta} = \mathbf{D}_{\alpha\beta\iota\lambda} \mathbf{K}_{\iota\lambda} \quad \mathbf{q}_{\alpha} = \mathbf{C}_{\alpha\beta} \boldsymbol{\gamma}_{\beta}$$

The fourth and second order tensor $\mathbf{D}_{\alpha\beta\iota\lambda}$ (bending stiffness) and $\mathbf{C}_{\alpha\beta}$ (shear stiffness) are symmetric, positive definite.

Constitutive equations

For an homogeneous, isotropic material (Greek indexes equal 1,2)

$$\mathbf{M}_{\alpha\beta} = \mathbf{D}_{\alpha\beta\iota\lambda} \mathbf{K}_{\iota\lambda} \quad \mathbf{q}_{\alpha} = \mathbf{C}_{\alpha\beta} \boldsymbol{\gamma}_{\beta}$$

The fourth and second order tensor $\mathbf{D}_{\alpha\beta\iota\lambda}$ (bending stiffness) and $\mathbf{C}_{\alpha\beta}$ (shear stiffness) are symmetric, positive definite.

The variables

$$\mathbf{K} := \text{Grad}(\boldsymbol{\theta}), \quad \boldsymbol{\gamma} := \text{grad}(w) - \boldsymbol{\theta}.$$

are the bending curvature and shear strain. The symmetric gradient of a vector field is defined as

$$\text{Grad}(\boldsymbol{\theta}) := \frac{1}{2} (\nabla \boldsymbol{\theta} + \nabla^T \boldsymbol{\theta}).$$

The kinetic and potential energy density \mathcal{K} and \mathcal{U} read

$$\mathcal{K} = \frac{1}{2} \left\{ \rho h \left(\frac{\partial w}{\partial t} \right)^2 + \frac{\rho h^3}{12} \frac{\partial \boldsymbol{\theta}}{\partial t} \cdot \frac{\partial \boldsymbol{\theta}}{\partial t} \right\},$$
$$\mathcal{U} = \frac{1}{2} \{ \boldsymbol{M} : \boldsymbol{K} + \boldsymbol{q} \cdot \boldsymbol{\gamma} \},$$

where $\boldsymbol{M} : \boldsymbol{K} := \sum_{\alpha, \beta} m_{\alpha\beta} \kappa_{\alpha\beta}$ is the tensor contraction.
The Hamiltonian (total energy) is the sum of the two

$$H = \int_{\Omega} (\mathcal{K} + \mathcal{U}) \, \mathrm{d}\Omega.$$

Energy, coenergy variables

The choice of the energy variables is the same as in¹

$$\begin{aligned}\alpha_w &= \rho h \frac{\partial w}{\partial t}, & \alpha_\theta &= \frac{\rho h^3}{12} \frac{\partial \theta}{\partial t}, \\ \mathbf{A}_\kappa &= \mathbf{K}, & \alpha_\gamma &= \gamma.\end{aligned}$$

¹A. Macchelli, C. Melchiorri, and L. Bassi. “Port-based Modelling and Control of the Mindlin Plate”. In: *Proceedings of the 44th IEEE Conference on Decision and Control*. 2005, pp. 5989–5994.

Energy, coenergy variables

The choice of the energy variables is the same as in¹

$$\begin{aligned}\alpha_w &= \rho h \frac{\partial w}{\partial t}, & \alpha_\theta &= \frac{\rho h^3}{12} \frac{\partial \theta}{\partial t}, \\ \mathbf{A}_\kappa &= \mathbf{K}, & \alpha_\gamma &= \gamma.\end{aligned}$$

The coenergies are given by the Hamiltonian variational derivative

$$\begin{aligned}e_w &:= \frac{\delta H}{\delta \alpha_w} = \frac{\partial w}{\partial t}, & e_\theta &:= \frac{\delta H}{\delta \alpha_\theta} = \frac{\partial \theta}{\partial t}, \\ \mathbf{E}_\kappa &:= \frac{\delta H}{\delta \mathbf{A}_\kappa} = \mathbf{M}, & e_{\epsilon_s} &:= \frac{\delta H}{\delta \alpha_\gamma} = \mathbf{q}.\end{aligned}$$

¹A. Macchelli, C. Melchiorri, and L. Bassi. “Port-based Modelling and Control of the Mindlin Plate”. In: *Proceedings of the 44th IEEE Conference on Decision and Control*. 2005, pp. 5989–5994.

The port-Hamiltonian system is expressed as follows

$$\frac{\partial}{\partial t} \begin{pmatrix} \alpha_w \\ \boldsymbol{\alpha}_\theta \\ \mathbf{A}_\kappa \\ \boldsymbol{\alpha}_\gamma \end{pmatrix} = \underbrace{\begin{bmatrix} 0 & 0 & 0 & \text{div} \\ 0 & 0 & \text{Div} & \mathbf{I}_{2 \times 2} \\ 0 & \text{Grad} & 0 & 0 \\ \text{grad} & -\mathbf{I}_{2 \times 2} & 0 & 0 \end{bmatrix}}_J \begin{pmatrix} e_w \\ \mathbf{e}_\theta \\ \mathbf{E}_\kappa \\ e_\gamma \end{pmatrix},$$

with J skew symmetric. Moreover

$$\begin{pmatrix} e_w \\ \mathbf{e}_\theta \\ \mathbf{E}_\kappa \\ e_\gamma \end{pmatrix} = \underbrace{\begin{bmatrix} 1/(\rho h) & 0 & 0 & 0 \\ 0 & 12/(\rho h^3) & 0 & 0 \\ 0 & 0 & \mathbf{D} & 0 \\ 0 & 0 & 0 & \mathbf{C} \end{bmatrix}}_Q \begin{pmatrix} \alpha_w \\ \boldsymbol{\alpha}_\theta \\ \mathbf{A}_\kappa \\ \boldsymbol{\alpha}_\gamma \end{pmatrix},$$

with Q coercive.

Boundary variables

Taking the energy rate and applying of the Green theorem

$$\dot{H} = \int_{\partial\Omega} \{ w_t q_n + \omega_n m_{nn} + \omega_s m_{ns} \} \, ds.$$

The dynamic boundary variable are defined as

Shear Force	$q_n := \mathbf{e}_\gamma \cdot \mathbf{n},$
Flexural momentum	$m_{nn} := \mathbf{E}_\kappa : (\mathbf{n} \otimes \mathbf{n}),$
Torsional momentum	$m_{ns} := \mathbf{E}_\kappa : (\mathbf{s} \otimes \mathbf{n}),$

where $\mathbf{u} \otimes \mathbf{v}$ denotes the outer product of vectors.

The corresponding power conjugated (and essential) boundary variables are

Vertical velocity	$w_t := e_w,$
Flexural rotation	$\omega_n := \mathbf{e}_\theta \cdot \mathbf{n},$
Torsional rotation	$\omega_s := \mathbf{e}_\theta \cdot \mathbf{s}.$

- 1 PH formulation of the Mindlin plate
- 2 Structure preserving discretization
 - Boundary control through forces and momenta
 - Boundary control through kinematic variables
- 3 Discretization procedure
- 4 Numerical simulations

Main step to follow

The structure-preserving discretization consists of three steps:

- ① write the system in weak form;
- ② perform integrations by parts to get the chosen boundary control;
- ③ select the finite element spaces to achieve a finite-dimensional system.

J decomposition

Decomposing J in operators to be integrated by parts

$$J = J_{\text{div}} + J_{\text{grad}} + J_I$$

$$J = \begin{bmatrix} 0 & 0 & 0 & \text{div} \\ 0 & 0 & \text{Div} & \mathbf{I}_{2 \times 2} \\ 0 & \text{Grad} & 0 & 0 \\ \text{grad} & -\mathbf{I}_{2 \times 2} & 0 & 0 \end{bmatrix}$$

J decomposition

Decomposing J in operators to be integrated by parts

$$J = J_{\text{div}} + J_{\text{grad}} + J_I$$

$$J_{\text{div}} := \begin{bmatrix} 0 & 0 & 0 & \text{div} \\ 0 & 0 & \text{Div} & 0 \\ 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$

J decomposition

Decomposing J in operators to be integrated by parts

$$J = J_{\text{div}} + J_{\text{grad}} + J_I$$

$$J_{\text{grad}} := \begin{bmatrix} 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 \\ 0 & \text{Grad} & 0 & 0 \\ \text{grad} & 0 & 0 & 0 \end{bmatrix}$$

J decomposition

Decomposing J in operators to be integrated by parts

$$J = J_{\text{div}} + J_{\text{grad}} + J_I$$

$$J_I := \begin{bmatrix} 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & \mathbf{I}_{2 \times 2} \\ 0 & 0 & 0 & 0 \\ 0 & -\mathbf{I}_{2 \times 2} & 0 & 0 \end{bmatrix}$$

J decomposition

Decomposing J in operators to be integrated by parts

$$J = J_{\text{div}} + J_{\text{grad}} + J_I$$

From these definitions, it holds

$$J_{\text{div}} = -J_{\text{grad}}^*,$$

where A^* is the formal adjoint of operator A .

To simplify the notation, all test and unknown functions can be collected in one set variable

$$\begin{aligned}v &:= (v_w, \mathbf{v}_\theta, \mathbf{V}_\kappa, \mathbf{v}_\gamma), \\ \alpha &:= (\alpha_w, \boldsymbol{\alpha}_\theta, \mathbf{A}_\kappa, \boldsymbol{\alpha}_\gamma), \\ e &:= (e_w, \mathbf{e}_\theta, \mathbf{E}_\kappa, \mathbf{e}_\gamma),\end{aligned}$$

so that the previous system is rewritten compactly as

$$\left(v, \frac{\partial \alpha}{\partial t}\right) = (v, Je).$$

If the operator J_{div} is integrated by parts then

$$(v, Je) = j_{\text{grad}}(v, e) + f_N(v). \quad (1)$$

The bilinear form

$$\begin{aligned} j_{\text{grad}}(v, e) &= (J_{\text{div}}^* v, e) + (v, J_{\text{grad}} e) + (v, J_I e), \\ &= (-J_{\text{grad}} v, e) + (v, J_{\text{grad}} e) + (v, J_I e), \end{aligned}$$

is skew symmetric.

Gradient formulation

If the operator J_{div} is integrated by parts then

$$(v, Je) = j_{\text{grad}}(v, e) + f_N(v). \quad (1)$$

The functional

$$\begin{aligned} f_N(v) &= \int_{\partial\Omega} \{v_w q_n + v_{\omega_n} m_{nn} + v_{\omega_s} m_{ns}\} \, ds, \\ &= \int_{\partial\Omega} v_{\partial} u_{\partial} \, ds. \end{aligned}$$

express the boundary control u_{∂} in terms of forces and momenta:

$$u_{\partial} = \text{Trace} \begin{pmatrix} q_n \\ m_{nn} \\ m_{ns} \end{pmatrix} \quad y_{\partial} = \text{Trace} \begin{pmatrix} w_t \\ \omega_n \\ \omega_s \end{pmatrix}.$$

If the operator J_{grad} is integrated by parts then

$$(v, Je) = j_{\text{div}}(v, e) + f_D(v), \quad (2)$$

The bilinear form

$$\begin{aligned} j_{\text{div}}(v, e) &= (v, J_{\text{div}} e) + (J_{\text{grad}}^* v, e) + (v, J_I e), \\ &= (v, J_{\text{div}} e) + (-J_{\text{div}} v, e) + (v, J_I e), \end{aligned}$$

is skew symmetric.

Divergence formulation

If the operator J_{grad} is integrated by parts then

$$(v, Je) = j_{\text{div}}(v, e) + f_D(v), \quad (2)$$

The functional

$$\begin{aligned} f_D(v) &= \int_{\partial\Omega} \{v_{q_n} w_t + v_{m_{nn}} \omega_n + v_{m_{ns}} \omega_s\} \, ds, \\ &= \int_{\partial\Omega} v_{\partial} u_{\partial} \, ds. \end{aligned}$$

expresses the boundary controls u_{∂} in terms of linear and angular velocities:

$$u_{\partial} = \text{Trace} \begin{pmatrix} w_t \\ \omega_n \\ \omega_s \end{pmatrix} \quad y_{\partial} = \text{Trace} \begin{pmatrix} q_n \\ m_{nn} \\ m_{ns} \end{pmatrix}_{\partial\Omega}.$$

Plan

- 1 PH formulation of the Mindlin plate
- 2 Structure preserving discretization
- 3 Discretization procedure
 - Finite-dimensional system
- 4 Numerical simulations

Homogeneous boundary conditions

Homogeneous boundary conditions:

- Clamped (C): $w_t = 0$, $\omega_n = 0$, $\omega_s = 0$;
- Simply supported hard (S): $w_t = 0$, $m_{nn} = 0$, $\omega_s = 0$;
- Free (F): $q_n = 0$, $m_{nn} = 0$, $m_{ns} = 0$.

The **gradient** formulation is adopted to discretize the system. This implies that

- variables in **Blue** are imposed weakly by setting $f_N(v) = 0$
- variables in **Red** have to be imposed strongly (by select a functional space that incorporates those or by introducing Lagrange multipliers)

Test and co-energy variables are discretized by a Galerkin Method, while energy variables are retrieved using the relation $\alpha = Q^{-1}e$. Replacing the approximated variables into the weak form

$$\begin{bmatrix} \mathbf{M} & 0 \\ 0 & 0 \end{bmatrix} \frac{d}{dt} \begin{pmatrix} \mathbf{e} \\ \boldsymbol{\lambda} \end{pmatrix} = \begin{pmatrix} \mathbf{J} & \mathbf{G}_D \\ -\mathbf{G}_D^T & 0 \end{pmatrix} \begin{pmatrix} \mathbf{e} \\ \boldsymbol{\lambda} \end{pmatrix} + \begin{pmatrix} \mathbf{B}_N \\ 0 \end{pmatrix} \mathbf{u}_N$$

- \mathbf{G}_D accounts for essential (Dirichlet) BCs;
- \mathbf{B}_N account for inhomogeneous natural (Neumann) BCs;

Plan

- 1 PH formulation of the Mindlin plate
- 2 Structure preserving discretization
- 3 Discretization procedure
- 4 Numerical simulations**

Eigenvalues

BCs	Mode	$N = 10$	$N = 20$	H-H	D-R
CCCC	$\hat{\omega}_{11}$	1.5999	1.5917	1.591	1.594
	$\hat{\omega}_{21}$	3.0615	3.0410	3.039	3.046
	$\hat{\omega}_{12}$	3.0615	3.0410	3.039	3.046
	$\hat{\omega}_{22}$	4.3161	4.2682	4.263	4.285
SSSS	$\hat{\omega}_{11}$	0.9324	0.9324	0.930	0.930
	$\hat{\omega}_{21}$	2.2227	2.2226	2.219	2.219
	$\hat{\omega}_{12}$	2.2227	2.2226	2.219	2.219
	$\hat{\omega}_{22}$	3.4142	3.3608	3.405	3.406
SCSC	$\hat{\omega}_{11}$	1.3111	1.3013	1.300	1.302
	$\hat{\omega}_{21}$	2.4155	2.3966	2.394	2.398
	$\hat{\omega}_{12}$	2.9082	2.8871	2.885	2.888
	$\hat{\omega}_{22}$	3.8906	3.8458	3.839	3.852
CCCF	$\hat{\omega}_{\frac{1}{2}1}$	1.0855	1.0982	1.081	1.089
	$\hat{\omega}_{\frac{3}{2}1}$	1.7636	1.7461	1.744	1.758
	$\hat{\omega}_{\frac{1}{2}2}$	2.6696	2.6575	2.657	2.673
	$\hat{\omega}_{\frac{5}{2}1}$	3.2248	3.1997	3.197	3.216

Table: Eigenvalues for $h/L = 0.1$ using \mathbb{P}_1 :

■ reference, ■ $\varepsilon < 2\%$.

Eigenvalues

BCs	Mode	$N = 5$	$N = 10$	H-H	D-R
CCCC	$\hat{\omega}_{11}$	1.5976	1.5914	1.591	1.594
	$\hat{\omega}_{21}$	3.0584	3.0405	3.039	3.046
	$\hat{\omega}_{12}$	3.0677	3.0405	3.039	3.046
	$\hat{\omega}_{22}$	4.3109	4.2662	4.263	4.285
SSSS	$\hat{\omega}_{11}$	0.9304	0.9302	0.930	0.930
	$\hat{\omega}_{21}$	2.2223	2.2194	2.219	2.219
	$\hat{\omega}_{12}$	2.2224	2.2194	2.219	2.219
	$\hat{\omega}_{22}$	3.4128	3.4061	3.405	3.406
SCSC	$\hat{\omega}_{11}$	1.3053	1.3004	1.300	1.302
	$\hat{\omega}_{21}$	2.4040	2.3946	2.394	2.398
	$\hat{\omega}_{12}$	2.9060	2.8858	2.885	2.888
	$\hat{\omega}_{22}$	3.8721	3.8415	3.839	3.852
CCCF	$\hat{\omega}_{\frac{1}{2}1}$	1.0845	1.0797	1.081	1.089
	$\hat{\omega}_{\frac{3}{2}1}$	1.7559	1.7425	1.744	1.758
	$\hat{\omega}_{\frac{1}{2}2}$	2.6762	2.6547	2.657	2.673
	$\hat{\omega}_{\frac{5}{2}1}$	3.2186	3.1954	3.197	3.216

Table: Eigenvalues for $h/L = 0.1$ using \mathbb{P}_2 :

reference, $\epsilon < 2\%$, $\epsilon < 5\%$, $\epsilon < 15\%$.

Eigenvalues

BCs	Mode	$N = 10$	$N = 20$	H-H	D-R
CCCC	$\hat{\omega}_{11}$	0.1967	0.1765	0.1754	0.1754
	$\hat{\omega}_{21}$	0.4030	0.3604	0.3574	0.3576
	$\hat{\omega}_{12}$	0.4030	0.3604	0.3574	0.3576
	$\hat{\omega}_{22}$	0.6431	0.5358	0.5264	0.5274
SSSS	$\hat{\omega}_{11}$	0.1706	0.1128	0.0963	0.0963
	$\hat{\omega}_{21}$	0.3576	0.2660	0.2406	0.2406
	$\hat{\omega}_{12}$	0.3576	0.2660	0.2406	0.2406
	$\hat{\omega}_{22}$	0.5803	0.4442	0.3847	0.3848
SCSC	$\hat{\omega}_{11}$	0.1864	0.1487	0.1411	0.1411
	$\hat{\omega}_{21}$	0.3649	0.2829	0.2668	0.2668
	$\hat{\omega}_{12}$	0.3987	0.3485	0.3377	0.3377
	$\hat{\omega}_{22}$	0.6075	0.4933	0.4604	0.4608
CCCF	$\hat{\omega}_{\frac{1}{2}1}$	0.1238	0.1166	0.1166	0.1171
	$\hat{\omega}_{\frac{3}{2}1}$	0.2207	0.1954	0.1949	0.1951
	$\hat{\omega}_{\frac{1}{2}2}$	0.3204	0.3078	0.3080	0.3093
	$\hat{\omega}_{\frac{5}{2}1}$	0.4144	0.3751	0.3736	0.3740

Table: Eigenvalues for $h/L = 0.01$ using \mathbb{P}_1 :

reference, $\varepsilon < 2\%$, $\varepsilon < 5\%$, $\varepsilon < 15\%$, $\varepsilon < 30\%$, $\varepsilon < 50\%$.

Eigenvalues

BCs	Mode	$N = 5$	$N = 10$	H-H	D-R
CCCC	$\hat{\omega}_{11}$	0.1872	0.1762	0.1754	0.1754
	$\hat{\omega}_{21}$	0.3725	0.3598	0.3574	0.3576
	$\hat{\omega}_{12}$	0.4055	0.3598	0.3574	0.3576
	$\hat{\omega}_{22}$	0.6043	0.5335	0.5264	0.5274
SSSS	$\hat{\omega}_{11}$	0.0963	0.0963	0.0963	0.0963
	$\hat{\omega}_{21}$	0.2422	0.2406	0.2406	0.2406
	$\hat{\omega}_{12}$	0.2430	0.2406	0.2406	0.2406
	$\hat{\omega}_{22}$	0.3874	0.3848	0.3847	0.3848
SCSC	$\hat{\omega}_{11}$	0.1492	0.1418	0.1411	0.1411
	$\hat{\omega}_{21}$	0.2827	0.2683	0.2668	0.2668
	$\hat{\omega}_{12}$	0.3608	0.3394	0.3377	0.3377
	$\hat{\omega}_{22}$	0.4940	0.4654	0.4604	0.4608
CCCF	$\hat{\omega}_{\frac{1}{2}1}$	0.1197	0.1169	0.1166	0.1171
	$\hat{\omega}_{\frac{3}{2}1}$	0.2092	0.1960	0.1949	0.1951
	$\hat{\omega}_{\frac{1}{2}2}$	0.3188	0.3089	0.3080	0.3093
	$\hat{\omega}_{\frac{5}{2}1}$	0.3938	0.3757	0.3736	0.3740

Table: Eigenvalues for $h/L = 0.01$ using \mathbb{P}_2 :

reference. $\epsilon < 2\%$. $\epsilon < 5\%$. $\epsilon < 15\%$.

Thank you for your attention. Questions?