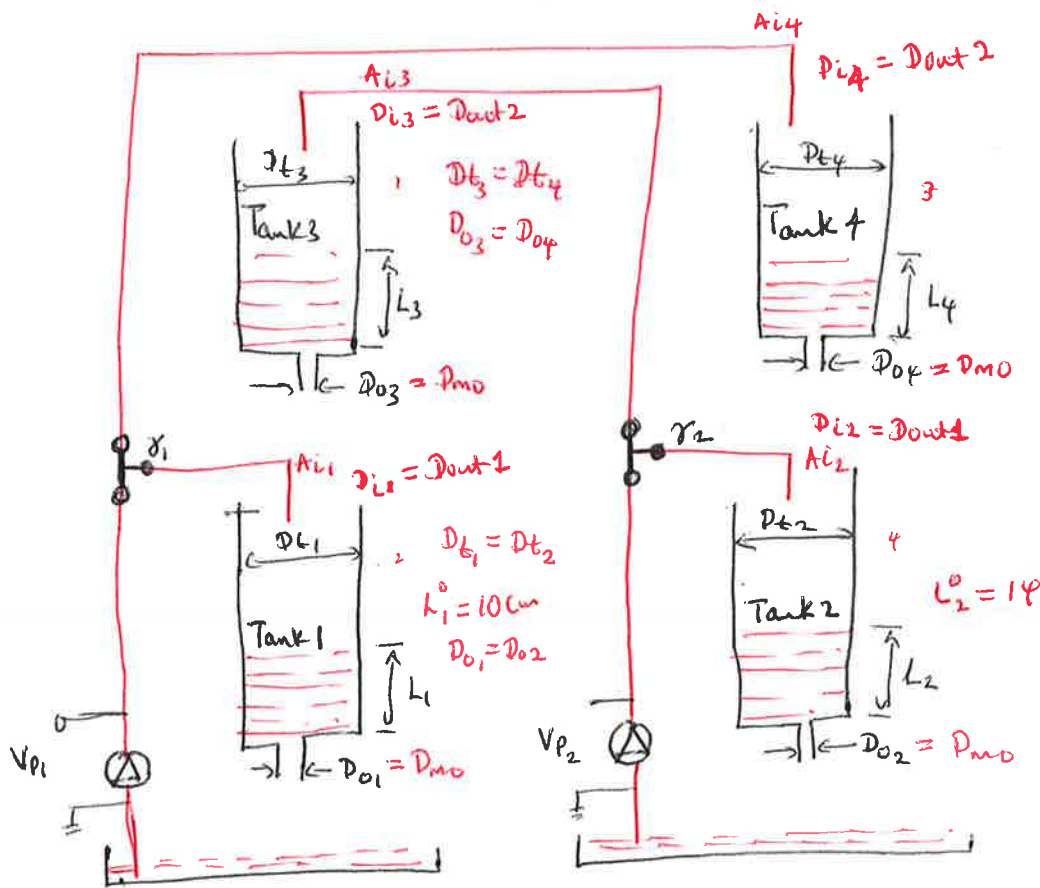


Quadruple Tank System



A_{ti} - Cross section of Tank i

a_{oi} - Cross-section of the outlet pipe for tank i

L_i - water level

A_{i1} - Tank 1 inlet area

Equation of motion

volumetric flow rate (out)

$$A_{ti} \frac{dL_i}{dt} = F_{mi} - f_{out i}$$

Rate of change of volume in tank i (up arrow to $\frac{dL_i}{dt}$)

Volumetric flow rate (in) (up arrow to F_{mi})

flow velocity \times cross-sectional area = volumetric flow rate

$$v_i = \sqrt{2gL_i}$$

$$A_{ti} = \frac{\pi D_{ti}^2}{4}$$

$$v_i A_{ti}$$

$$A_{t1} \frac{dL_1}{dt} = a_{o3} \sqrt{2gL_3} + \gamma_1 K_p V_{p1} - a_{o1} \sqrt{2gL_1}$$

$$A_{t2} \frac{dL_2}{dt} = a_{o4} \sqrt{2gL_4} + \gamma_2 K_p V_{p2} - a_{o2} \sqrt{2gL_2}$$

$$A_{t3} \frac{dL_3}{dt} = (1 - \gamma_2) K_p V_{p2} - a_{o3} \sqrt{2gL_3}$$

$$A_{t4} \frac{dL_4}{dt} = (1 - \gamma_1) K_p V_{p1} - a_{o4} \sqrt{2gL_4}$$

The control target is to control the water levels in the lower two tanks with the two pumps.

K_p - Pump flow constant

K_L - Tank water level sensor sensitivity

In short form:

$$\frac{dL_1}{dt} = -\frac{a_1}{A_1} \sqrt{2g} L_1 + \frac{a_3}{A_1} \sqrt{2g} L_3 + \frac{\gamma_1 K_1 V_{p1}}{A_1}$$

$$\frac{dL_2}{dt} = -\frac{a_2}{A_2} \sqrt{2g} L_2 + \frac{a_4}{A_2} \sqrt{2g} L_4 + \frac{\gamma_2 K_2 V_{p2}}{A_2}$$

$$\frac{dL_3}{dt} = -\frac{a_3}{A_3} \sqrt{2g} L_3 + \frac{(1-\gamma_2) K_2 V_{p2}}{A_3}$$

$$\frac{dL_4}{dt} = -\frac{a_4}{A_4} \sqrt{2g} L_4 + \frac{(1-\gamma_1) K_1 V_{p1}}{A_4}$$

Linearization

Define $x_i = L_i - L_i^0$ and $u_i = V_{pi} - V_i^0$

Using truncated Taylor's Series expansion

$$\sqrt{2g} L_i \approx \sqrt{2g} L_i^0 + \left. \frac{g}{\sqrt{2g} L_i} \right|_{L_i=L_i^0} (L_i - L_i^0) = \sqrt{2g} L_i^0 + \frac{g}{\sqrt{2g} L_i^0} x_i$$

At equilibrium, we have for tank 1 and tank 2

$$-\frac{a_1}{A_1} \sqrt{2g} L_1^0 + \frac{a_3}{A_1} \sqrt{2g} L_3^0 + \frac{\gamma_1 K_1 V_{p1}^0}{A_1} = 0$$

and for tank 3 and 4, we have,

$$-\frac{a_i}{A_i} \sqrt{2g} L_i^0 + \frac{(1-\gamma_j) K_j V_{pj}^0}{A_i} = 0 \quad \text{where } j=1 \text{ for tank 4 and } j=2 \text{ for tank 3}$$

we have

$$\dot{x}_1 = -\frac{a_1}{A_1} \sqrt{\frac{g}{2L_1^0}} x_1 + \frac{a_3}{A_1} \sqrt{\frac{g}{2L_3^0}} x_3 + \frac{\gamma_1 K_1 u_1}{A_1}$$

$$\dot{x}_3 = -\frac{a_3}{A_3} \sqrt{\frac{g}{2L_3^0}} x_3 + \frac{(-\gamma_2) K_2 u_2}{A_3}$$

$$\dot{x}_2 = -\frac{a_2}{A_2} \sqrt{\frac{g}{2L_2^0}} x_2 + \frac{a_4}{A_2} \sqrt{\frac{g}{2L_4^0}} x_4 + \frac{\gamma_2 K_2 u_2}{A_2}$$

$$\dot{x}_4 = -\frac{a_4}{A_4} \sqrt{\frac{g}{2L_4^0}} x_4 + \frac{(1-\gamma_1) K_1 u_1}{A_4}$$

Equilibrium Condition

$$\frac{dL_3}{dt} = -\frac{a_3}{A_3} \sqrt{2gL_3} + \frac{(1-\gamma_2)K_P V_{P_1}}{A_3}$$

at equilibrium

$$0 = -\frac{a_3}{A_3} \sqrt{2gL_3^0} + \frac{(1-\gamma_2)K_P V_{P_1}^0}{A_3}$$

$$a_3 \sqrt{2gL_3^0} = \frac{(1-\gamma_2)K_P V_{P_1}^0}{A_3} \Rightarrow V_{P_1}^0 = \frac{a_3 \sqrt{2gL_3^0}}{(1-\gamma_2)K_P} \text{ or } L_3^0 = \left[\frac{(1-\gamma_2)K_P V_{P_1}^0}{a_3} \right]^2 / 2g$$

(1)

$$a_3 \sqrt{2gL_3^0} = (1-\gamma_2)K_P V_{P_1}^0$$

Also,

$$\frac{dL_4}{dt} = -\frac{a_4}{A_4} \sqrt{2gL_4} + \frac{(1-\gamma_1)K_P V_{P_1}}{A_4}$$

at equilibrium.

$$0 = -\frac{a_4}{A_4} \sqrt{2gL_4^0} + \frac{(1-\gamma_1)K_P V_{P_1}^0}{A_4}$$

$$V_{P_1}^0 = \frac{a_4 \sqrt{2gL_4^0}}{(1-\gamma_1)K_P} \text{ or } \sqrt{2gL_4^0} = \frac{(1-\gamma_1)K_P V_{P_1}^0}{a_4} \Rightarrow L_4^0 = \left[\frac{(1-\gamma_1)K_P V_{P_1}^0}{a_4} \right]^2 / 2g$$

(2)

$$a_4 \sqrt{2gL_4^0} = (1-\gamma_1)K_P V_{P_1}^0$$

$$\frac{dL_1}{dt} = -\frac{a_1}{A_1} \sqrt{2gL_1} + \frac{a_3}{A_1} \sqrt{2gL_3} + \frac{\gamma_1 K_P V_{P_1}}{A_1}$$

at equilibrium

$$0 = -\frac{a_1}{A_1} \sqrt{2gL_1^0} + \frac{a_3}{A_1} \sqrt{2gL_3^0} + \frac{\gamma_1 K_P V_{P_1}^0}{A_1}$$

(3)

finally

$$\frac{dL_2}{dt} = -\frac{a_2}{A_2} \sqrt{2gL_2} + \frac{a_4}{A_2} \sqrt{2gL_4} + \frac{\gamma_2 K_P V_{P_2}}{A_2}$$

at equilibrium

$$0 = -\frac{a_2}{A_2} \sqrt{2gL_2^0} + \frac{a_4}{A_2} \sqrt{2gL_4^0} + \frac{\gamma_2 K_P V_{P_2}^0}{A_2}$$

(4)

from (3) and (1), we have

$$0 = \frac{-a_1 \sqrt{2g} L_1^0}{A_1} + \frac{(1-\gamma_2) K_p V_{p_2}^0}{A_1} + \frac{\gamma_1 K_p V_{p_1}^0}{A_1}$$

from (4) and (2), we have

$$0 = \frac{-a_2 \sqrt{2g} L_2^0}{A_2} + \frac{(1-\gamma_1) K_p V_{p_1}^0}{A_2} + \frac{\gamma_2 K_p V_{p_2}^0}{A_2}$$

Two equations, two unknowns

$$\left. \begin{aligned} \gamma_1 K_p V_{p_1}^0 + (1-\gamma_2) K_p V_{p_2}^0 &= a_1 \sqrt{2g} L_1^0 \\ (1-\gamma_1) K_p V_{p_1}^0 + \gamma_2 K_p V_{p_2}^0 &= a_2 \sqrt{2g} L_2^0 \end{aligned} \right\}$$

$$\left. \begin{aligned} V_{p_1}^0 + \frac{1-\gamma_2}{\gamma_1} V_{p_2}^0 &= \frac{a_1 \sqrt{2g} L_1^0}{\gamma_1 K_p} \\ V_{p_1}^0 + \frac{\gamma_2}{1-\gamma_1} V_{p_2}^0 &= \frac{a_2 \sqrt{2g} L_2^0}{(1-\gamma_1) K_p} \end{aligned} \right\}$$

$$\left[\frac{1-\gamma_2}{\gamma_1} - \frac{\gamma_2}{1-\gamma_1} \right] V_{p_2}^0 = \frac{a_1 \sqrt{2g} L_1^0}{\gamma_1 K_p} - \frac{a_2 \sqrt{2g} L_2^0}{(1-\gamma_1) K_p}$$

$$\left[\frac{(1-\gamma_2)(1-\gamma_1) - \gamma_1 \gamma_2}{\gamma_1 (1-\gamma_1)} \right] V_{p_2}^0 = \frac{a_1 \sqrt{2g} L_1^0}{\gamma_1 K_p} - \frac{a_2 \sqrt{2g} L_2^0}{(1-\gamma_1) K_p}$$

$$V_{p_2}^0 = \frac{a_1 (1-\gamma_1) \sqrt{2g} L_1^0 / K_p}{(1-\gamma_1)(1-\gamma_2) - \gamma_1 \gamma_2} - \frac{a_2 \gamma_1 \sqrt{2g} L_2^0 / K_p}{(1-\gamma_1)(1-\gamma_2) - \gamma_1 \gamma_2}$$

or

$$V_{p_1}^0 = \frac{a_1 \sqrt{2g} L_1^0}{\gamma_1 K_p} - \frac{1-\gamma_2}{\gamma_1} V_{p_2}^0$$

Then L_3^0 and L_4^0 are recovered from (1) and (2) as:

$$L_3^0 = \left[\frac{(1-\gamma_2) K_p V_{p_2}^0}{a_3} \right] / 2g \quad \text{and} \quad L_4^0 = \left[\frac{(1-\gamma_1) K_p V_{p_1}^0}{a_4} \right] / 2g$$

$$\gamma_1 = \frac{A_{i1}}{A_{i1} + A_{i4}}$$

$$\gamma_2 = \frac{A_{i2}}{A_{i2} + A_{i3}}$$

$$K_p = 3.3 \text{ cm}^3/\text{s}/\text{V}$$

$$K_L = 25/4.15 \text{ cm}/\text{V}$$

$$g = 981 \text{ cm}/\text{s}^2$$

$$\alpha_1 = \frac{0.3167}{0.3167 + 0.1781} = 0.6401$$

$$\alpha_2 = \frac{0.3167}{0.3167 + 0.1781} = 0.6401$$

$$a_1 = \frac{\pi \bar{D}_{o1}^2}{4} = \frac{\pi \bar{D}_{m0}^2}{4} = \frac{\pi (0.47625)^2}{4} = 0.1781 \text{ cm}^2$$

$$a_2 = \frac{\pi \bar{D}_{o2}^2}{4} = \frac{\pi \bar{D}_{m0}^2}{4} = 0.1781 \text{ cm}^2$$

$$a_3 = \frac{\pi \bar{D}_{o3}^2}{4}$$

$$a_4 = \frac{\pi \bar{D}_{o4}^2}{4}$$

$$A_{i3} = \frac{\pi \bar{D}_{i3}^2}{4} = \frac{\pi \bar{D}_{out2}^2}{4} = 0.1781 \text{ cm}^2$$

$$A_{i4} = \frac{\pi \bar{D}_{i4}^2}{4} = \frac{\pi \bar{D}_{out2}^2}{4} = 0.1781 \text{ cm}^2$$

$$A_{i1} - \text{Tank 1 inlet Area (cm}^2) = \frac{\pi \bar{D}_{i1}^2}{4} = \frac{\pi \bar{D}_{out1}^2}{4} = 0.3167 \text{ cm}^2$$

$$A_{i2} - \text{Tank 2 inlet Area (cm}^2) = \frac{\pi \bar{D}_{i2}^2}{4} = \frac{\pi \bar{D}_{out1}^2}{4} = 0.3167 \text{ cm}^2$$

$$\bar{D}_{out1} - \text{Out 1" Orifice Diameter (cm)} = 0.635 \text{ cm}$$

$$\bar{D}_{out2} - \text{Out 2" Orifice Diameter (cm)} = 0.47625 \text{ cm}$$

In compact form:

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} -\frac{1}{T_1} & 0 & \frac{A_3}{A_1 T_3} & 0 \\ 0 & -\frac{1}{T_2} & 0 & 0 \\ 0 & 0 & -\frac{1}{T_3} & 0 \\ 0 & 0 & 0 & -\frac{1}{T_4} \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + \begin{bmatrix} \frac{\gamma_1 K_1}{A_1} & 0 \\ 0 & \frac{\gamma_2 K_2}{A_2} \\ 0 & \frac{(1-\gamma_2)K_2}{A_3} \\ \frac{(1-\gamma_1)K_1}{A_4} & 0 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$

$$\begin{bmatrix} y_1 \\ y_2 \end{bmatrix} = \begin{bmatrix} K_{L1} & 0 & 0 & 0 \\ 0 & K_{L2} & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix}$$

where

K_{Li} - Tank i water level sensor sensitivity.

$$T_i = \frac{A_i}{a_i} \sqrt{\frac{2i_1^0}{g}} \quad ; i=1, \dots, 4.$$

In transfer function

$$sX_1 = -\frac{1}{T_1} X_1 + \frac{A_3}{A_1 T_3} X_3 + \frac{\gamma_1 K_1}{A_1} U_1 \Rightarrow X_1 = \frac{A_3/A_1 T_3}{s + \frac{1}{T_1}} X_3 + \frac{\gamma_1 K_1/A_1}{s + \frac{1}{T_1}} U_1$$

$$sX_2 = -\frac{1}{T_2} X_2 + \frac{A_4}{A_2 T_4} X_4 + \frac{\gamma_2 K_2}{A_2} U_2 \Rightarrow X_2 = \frac{A_4/A_2 T_4}{s + \frac{1}{T_2}} X_4 + \frac{\gamma_2 K_2/A_2}{s + \frac{1}{T_2}} U_2$$

$$sX_3 = -\frac{1}{T_3} X_3 + \frac{(1-\gamma_2)K_2}{A_3} U_2 \Rightarrow X_3 = \frac{(1-\gamma_2)K_2/A_3}{s + \frac{1}{T_3}} U_2$$

$$sX_4 = -\frac{1}{T_4} X_4 + \frac{(1-\gamma_1)K_1}{A_4} U_1 \Rightarrow X_4 = \frac{(1-\gamma_1)K_1/A_4}{s + \frac{1}{T_4}} U_1$$

\Downarrow

$$X_1 = \frac{(1-\gamma_2)K_2/A_1 T_3}{(s + \frac{1}{T_1})(s + \frac{1}{T_3})} U_2 + \frac{\gamma_1 K_1/A_1}{s + \frac{1}{T_1}} U_1$$

$$X_2 = \frac{(1-\gamma_1)K_1/A_2 T_4}{(s + \frac{1}{T_2})(s + \frac{1}{T_4})} U_1 + \frac{\gamma_2 K_2/A_2}{s + \frac{1}{T_2}} U_2$$

This yields

$$Y_1(s) = \frac{\gamma_1 K_1 K_{L1} T_1 / A_1}{T_1 s + 1} U_1(s) + \frac{(1-\gamma_2) K_2 K_{L1} T_1 / A_1}{(T_1 s + 1)(T_3 s + 1)} U_2(s)$$

$$Y_2(s) = \frac{(1-\gamma_1) K_1 K_{L2} T_2 / A_2}{(T_2 s + 1)(T_4 s + 1)} U_1(s) + \frac{\gamma_2 K_2 K_{L2} T_2 / A_2}{T_2 s + 1} U_2(s)$$

$$\begin{bmatrix} Y_1(s) \\ Y_2(s) \end{bmatrix} = \begin{bmatrix} \frac{\gamma_1 C_1}{T_1 s + 1} & \frac{(1-\gamma_2) C_2}{(T_1 s + 1)(T_3 s + 1)} \\ \frac{(1-\gamma_1) C_3}{(T_2 s + 1)(T_4 s + 1)} & \frac{\gamma_2 C_4}{T_2 s + 1} \end{bmatrix} \begin{bmatrix} U_1(s) \\ U_2(s) \end{bmatrix}$$

where $C_1 = K_1 K_{L1} T_1 / A_1$, $C_2 = K_2 K_{L1} T_1 / A_1$
 $C_3 = K_1 K_{L2} T_2 / A_2$, $C_4 = K_2 K_{L2} T_2 / A_2$

Note that if $K_1 = K_2$
 and $K_{L1} = K_{L2}$, then
 $C_1 = C_2$ and $C_3 = C_4$.

Torsion Constant

$$T_1 = \frac{A_1}{a_1} \sqrt{\frac{2 L_1^0}{g}} \quad \text{where} \quad A_1 = \frac{\pi D_{L1}^2}{4} = \frac{\pi \times (4.445)^2}{4} \text{ cm}^2 = 15.518 \text{ cm}^2$$

$$= \frac{15.518}{0.178} \sqrt{\frac{2 \times 10}{981}} = 15.25 \text{ sec.} \quad L_1^0 = 10 \text{ cm}$$

$$a_1 = \frac{\pi D_{M0}^2}{4} = \frac{\pi \times (0.47625)^2}{4} \text{ cm}^2 = 0.178 \text{ cm}^2$$

$$L_1^0 = 10 \text{ cm} \quad g = 981 \text{ cm/s}^2$$

$$T_2 = \frac{A_2}{a_2} \sqrt{\frac{2 L_2^0}{g}} \quad \text{where} \quad A_2 = \frac{\pi D_{L2}^2}{4} = \frac{\pi \times (4.445)^2}{4} \text{ cm}^2 = 15.518 \text{ cm}^2$$

$$= \frac{15.518}{0.178} \sqrt{\frac{2 \times 15}{981}} = 14.724 \text{ sec.} \quad L_2^0 = 14 \text{ cm}$$

$$a_2 = \frac{\pi D_{M0}^2}{4} = \frac{\pi \times (0.47625)^2}{4} \text{ cm}^2 = 0.178 \text{ cm}^2$$

Here

$$L_3^0 = \left[\frac{(1-\gamma_2) a_1 (1-\gamma_1) \sqrt{2g} L_1^0}{a_3 [(1-\gamma_1)(1-\gamma_2) - \gamma_1 \gamma_2]} - \frac{(1-\gamma_2) a_2 \gamma_1 \sqrt{2g} L_2^0}{a_3 [(1-\gamma_1)(1-\gamma_2) - \gamma_1 \gamma_2]} \right]^2 / 2g$$

$1 - \gamma_1 - \gamma_2 + \gamma_1 \gamma_2 - \gamma_1 \gamma_2$

$$L_3^2 = \left[\frac{a_1 (1-\gamma_1 - \gamma_2 + \gamma_1 \gamma_2) \sqrt{2g} L_1^0 - a_2 (\gamma_1 - \gamma_1 \gamma_2) \sqrt{2g} L_2^0}{a_3 [1 - \gamma_1 - \gamma_2]} \right]^2 / 2g$$

Setting $L_{10} = 10 \text{ cm}$ and $L_{20} = 14 \text{ cm}$, we computed L_{30} and L_{40} as

$$L_{30} = 2.6088 \text{ cm}$$

$$L_{40} = ~~10.5196~~ 0.7573 \text{ cm}$$

Also

$$V_{p10} = ~~21.5423~~ 5.7861 \text{ V}$$

$$V_{p20} = 10.7278$$

The open-loop transfer function is given by

$$G(s) = \begin{bmatrix} \frac{\gamma_1 C_1}{T_1 s + 1} & \frac{(1-\gamma_2) C_2}{(T_1 s + 1)(T_3 s + 1)} \\ \frac{(1-\gamma_1) C_3}{(T_2 s + 1)(T_4 s + 1)} & \frac{\gamma_2 C_4}{T_2 s + 1} \end{bmatrix} \quad \text{where} \quad \gamma_1 = \frac{A_{i1}}{A_{i1} + A_{i4}}$$

$$\gamma_2 = \frac{A_{i2}}{A_{i2} + A_{i3}}$$

$$C_1 = K_1 K_{L1} T_1 / A_1, \quad C_2 = K_2 K_{L1} T_1 / A_1, \quad C_3 = K_1 K_{L2} T_2 / A_2$$

$$C_4 = K_2 K_{L2} T_2 / A_2$$

$$T_1 = \frac{A_1}{a_1} \sqrt{\frac{2L_1^0}{g}} \quad T_2 = \frac{A_2}{a_2} \sqrt{\frac{2L_2^0}{g}} \quad T_3 = \frac{A_3}{a_3} \sqrt{\frac{2L_3^0}{g}} \quad \text{and} \quad T_4 = \frac{A_4}{a_4} \sqrt{\frac{2L_4^0}{g}}$$

with $T_1 = 12.4381 \text{ sec}$

$T_2 = 14.7170 \text{ sec}$

$T_3 = 6.3529 \text{ sec}$

$T_4 = \cancel{10.7571} \text{ sec}$
 3.4229

$C_1 = 15.9341$

$C_2 = 15.9341$

$C_3 = 18.8534$

$C_4 = 18.8534$

$\gamma_1 = 0.64$

$\gamma_2 = 0.64$

which gives

$$G(s) = \begin{bmatrix} \frac{10.2}{12.44s+1} & \frac{\cancel{10.7571} 5.736}{(12.44s+1)(6.35s+1)} \\ \frac{\cancel{6.787} 6.787}{(14.72s+1)(\cancel{12.76} s+1)} & \frac{12.07}{14.72s+1} \end{bmatrix}$$

Internal model control

$$G(s)^{-1} = \begin{bmatrix} G(2,2) & -G(1,2) \\ -G(2,1) & G(1,1) \end{bmatrix} / \text{Det}(G)$$

$$\text{Det}(G) = \frac{10.2 \times 12.07}{(12.44s+1)(14.72s+1)} - \frac{6.787^2}{(12.44s+1)(6.35s+1)(14.72s+1)(12.76s+1)}$$

$$= \frac{123.114(6.35s+1)(12.76s+1) - 46.0634}{(12.44s+1)(14.72s+1)(6.35s+1)(12.76s+1)}$$

$$G(s)^{-1} = \begin{bmatrix} \frac{12.07}{14.72s+1} & \frac{-6.787}{(12.44s+1)(6.35s+1)} \\ \frac{-6.787}{(14.72s+1)(12.76s+1)} & \frac{10.2}{12.44s+1} \end{bmatrix} / \frac{123.114(6.35s+1)(12.76s+1) - 46.0634}{(12.44s+1)(14.72s+1)(6.35s+1)(12.76s+1)}$$

$$G(s)^{-1}$$

=

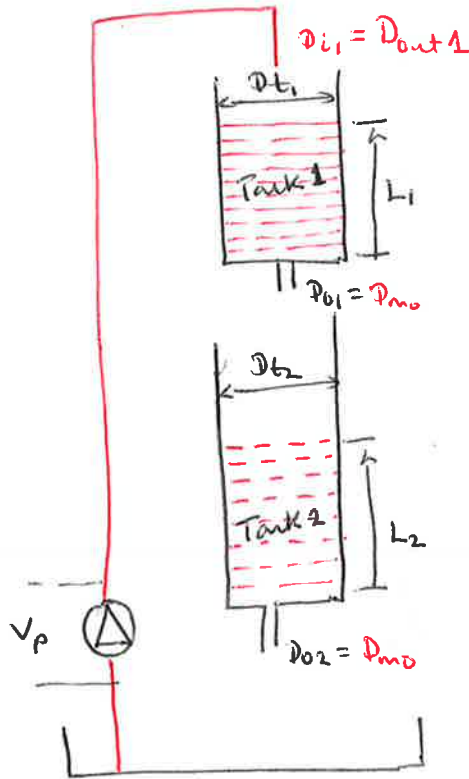
$$\left[\begin{array}{cc} (12.44s+1)(6.35s+1)(12.76s+1)(12.07) & -6.787(14.72s+1)(12.76s+1) \\ -6.787(12.44s+1)(6.35s+1) & 10.2(14.72s+1)(6.35s+1)(12.76s+1) \end{array} \right]$$

$$123.114(6.35s+1)(12.76s+1) - 46.0634$$

$$9975s^2 + 2353s + 77005$$

Coupled Tank System (Configuration #2)

(1)



A controller is designed to regulate or track the level in Tank 2.

The pump feeds into Tank 1 which in turn feeds into Tank 2.

Equation of motion

$$A_{t2} \frac{dL_2}{dt} = a_{01} \sqrt{2gL_1} - a_{02} \sqrt{2gL_2}$$

Linearized

Define $x_1 = L_1 - L_1^0$ and $x_2 = L_2 - L_2^0$

and using truncated Taylor's series expansion

$$\sqrt{2gL_i} \approx \sqrt{2gL_i^0} + \left. \frac{g}{\sqrt{2gL_i}} \right|_{L_i=L_i^0} (L_i - L_i^0) = \sqrt{2gL_i^0} + \frac{g}{\sqrt{2gL_i^0}} x_i$$

Here

$$A_{t2} \frac{d}{dt} (x_2 + L_2^0) = a_{01} \sqrt{2gL_1^0} + \frac{a_{01} g}{\sqrt{2gL_1^0}} x_1 - a_{02} \sqrt{2gL_2^0} - \frac{a_{02} g}{\sqrt{2gL_2^0}} x_2$$

or

$$A_{t2} \frac{dx_2}{dt} = a_{01} \sqrt{\frac{g}{2L_1^0}} x_1 - a_{02} \sqrt{\frac{g}{2L_2^0}} x_2 \Rightarrow \frac{dx_2}{dt} = \frac{a_{01}}{A_{t2}} \sqrt{\frac{g}{2L_1^0}} x_1 - \frac{a_{02}}{A_{t2}} \sqrt{\frac{g}{2L_2^0}} x_2$$

$$\text{Let } T_2 = \frac{A_{t2}}{a_{02}} \sqrt{\frac{2L_2^0}{g}} \text{ and } T_1 = \frac{A_{t1}}{a_{01}} \sqrt{\frac{2L_1^0}{g}}$$

Here we have

$$\dot{x}_2 = -\frac{1}{T_2} x_2 + \frac{A_1}{A_2 T_1} x_1$$

Taking Laplace transforms yields

$$s X_2(s) = -\frac{1}{T_2} X_2(s) + \frac{A_1}{A_2 T_1} X_1(s)$$

$$\left(s + \frac{1}{T_2}\right) X_2(s) = \frac{A_1}{A_2 T_1} X_1(s)$$

$$\frac{X_2(s)}{X_1(s)} = \frac{\frac{A_1}{A_2 T_1}}{s + \frac{1}{T_2}} \quad \text{or} \quad \frac{\frac{A_1}{A_2} \times \frac{T_2}{T_1}}{T_2 s + 1} = \frac{K_{dc-2}}{T_2 s + 1}$$

$$K_{dc-2} = \frac{A_1}{A_2} \times \left(\frac{A_2}{a_2} \sqrt{\frac{2L_2^0}{g}} \right) \times \left(\frac{a_1}{A_1} \sqrt{\frac{g}{2L_1^0}} \right) = \frac{a_1}{a_2} \sqrt{\frac{L_2^0}{L_1^0}}$$

Note: At equilibrium

$$0 = \frac{a_1}{A_1} \sqrt{\frac{g}{2L_1^0}} - \frac{a_2}{A_2} \sqrt{\frac{2L_2^0}{g}} \quad A_2 \frac{dL_2^0}{dt} = a_1 \sqrt{2gL_1^0} - a_2 \sqrt{2gL_2^0}$$

$$\sim a_2 \sqrt{2gL_2^0} = a_1 \sqrt{2gL_1^0}$$

$$\text{and } L_1^0 = \left(\frac{a_2}{a_1} \right)^2 L_2^0 = \left(\frac{0.1781 \text{ cm}^2}{0.1481 \text{ cm}^2} \right)^2 \times 15 \text{ cm} = 15 \text{ cm}$$

$$K_{dc-2} = \left(\frac{a_1}{a_2} \right) \sqrt{\frac{L_2^0}{L_1^0}} = \left(\frac{0.1781 \text{ cm}^2}{0.1481 \text{ cm}^2} \right) \times \sqrt{\frac{15 \text{ cm}}{15 \text{ cm}}} = 1$$

$$T_2 = \frac{A_2}{a_2} \sqrt{\frac{2L_2^0}{g}} = \left(\frac{15.5179 \text{ cm}^2}{0.1781 \text{ cm}^2} \right) \sqrt{\frac{2 \times 15 \text{ cm}}{981 \text{ cm/s}^2}} = 15.2335 \text{ s}$$

Hence

$$\frac{X_2(s)}{X_1(s)} = \frac{1}{15.2s + 1}$$

Now we derive the dynamics of tank 1 with respect to the tank level and the input voltage.

Equation of motion

$$At_1 \frac{dL_1}{dt} = K_p V_p - a_1 \sqrt{2gL_1}$$

Linearized

$$A_1 \frac{d(x_1 + L_1^0)}{dt} = K_p (u + V_p^0) - a_1 \sqrt{2gL_1^0} - \frac{a_1 g}{\sqrt{2gL_1^0}} x_1$$

$$\text{or}$$

$$A_1 \frac{dx_1}{dt} = K_p u + K_p V_p^0 - a_1 \sqrt{2gL_1^0} - a_1 \sqrt{\frac{g}{2L_1^0}} x_1$$

$$\Rightarrow A_1 \frac{dx_1}{dt} = K_p u - a_1 \sqrt{\frac{g}{2L_1^0}} x_1$$

or

$$\frac{dx_1}{dt} = -\frac{a_1}{A_1} \sqrt{\frac{g}{2L_1^0}} x_1 + \frac{K_p}{A_1} u$$

or

$$\dot{x}_1 = -\frac{1}{T_1} x_1 + \frac{K_p}{A_1} u$$

$$sX_1(s) = -\frac{1}{T_1} X_1(s) + \frac{K_p}{A_1} U(s)$$

$$\text{or} \quad \frac{X_1(s)}{U(s)} = \frac{K_p/A_1}{s + \frac{1}{T_1}} = \frac{K_p T_1/A_1}{T_1 s + 1} =$$

$$\text{or } \frac{X_1(s)}{U(s)} = \frac{K_{dc-1}}{T_1 s + 1}$$

$$T_1 = \frac{A_1}{a_1} \sqrt{\frac{2L_1^0}{g}} = \left(\frac{15.5179 \text{ cm}^2}{0.1781 \text{ cm}^2} \right) \times \sqrt{\frac{2 \times 15 \text{ cm}}{981 \text{ cm/s}^2}} = 15.2335 \text{ s}$$

$$K_{dc-1} = \frac{K_p \times T_1}{A_1} = \frac{3.3 \text{ cm}^3/\text{s/V} \times 15.2335 \text{ s}}{15.5179 \text{ cm}^2} = 3.2395 \text{ cm/V}$$

Put together Tank 2 dynamics Tank 1 dynamics

$$\begin{aligned} \frac{X_2(s)}{U(s)} &= \left[\frac{X_2(s)}{X_1(s)} \right] \times \left[\frac{X_1(s)}{U(s)} \right] = \frac{K_{dc-2}}{T_2 s + 1} \times \frac{K_{dc-1}}{T_1 s + 1} = \frac{K_{dc}}{(T_1 s + 1)(T_2 s + 1)} \\ &= \frac{3.2395}{(15.235s + 1)(15.238s + 1)} \end{aligned}$$

We can develop a feedback controller directly for the above transfer-function. However, in the quarter workbook, a controller is first developed for ~~the~~ Tank 1 such that ~~at~~ steady-state the ~~height~~ ^{level} in tank 1 is about the equilibrium level of 15 cm. ~~and then~~ ^{This is the inner loop controller.} Then a

Second controller is designed to regulate the level of tank 2. In this case the inner loop controller must be such that ~~the~~ tank 1 dynamics is much faster than that of tank 2.

Following the above strategy, I develop an IMC-based control as follows.

For tank 1.

$$\text{we have } G_1(s) = \frac{K_{dc-1}}{T_1 s + 1} = \frac{X_1(s)}{U(s)}$$

~~The~~ The internal model controller is obtained by

Inverting the plant.

(3)

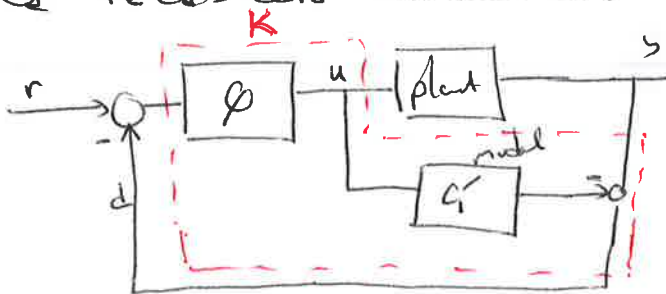
$$\text{Hence } \phi_0 = G^{-1} = \frac{T_1 s + 1}{K_{dc-1}} = \frac{15.2s + 1}{3.24} \leftarrow \text{This is improper}$$

Augmenting with a first order filter $f(s) = \frac{1}{\lambda s + 1}$

we have

$$\phi = \phi_0 f = \frac{15.2s + 1}{3.24} \times \frac{1}{\lambda s + 1} = \frac{15.2s + 1}{3.24(\lambda s + 1)}$$

Now using the connection between internal model controller and feedback controller:



$$\begin{aligned} u &= \phi(r-d) \text{ where } d = y - G u \\ &= \phi(r + G u - y) \\ &= \phi r + \phi G u - \phi y \\ &= \phi(r-y) + \phi G u \end{aligned}$$

$$\begin{aligned} \sim u - \phi G u &= \phi(r-y) \\ (1 - \phi G)u &= \phi(r-y) \end{aligned}$$

$$\sim u = \frac{\phi}{1 - \phi G} (r-y) \Leftrightarrow \text{This exactly like your feedback controller where}$$

$$u = K(r-y) \uparrow \text{err.}$$

Hence

$$K(s) = \frac{\phi(s)}{1 - \phi G(s)}$$

$$\text{But } \phi G = \frac{15.2s + 1}{3.24(\lambda s + 1)} \times \frac{3.24}{15.2s + 1} = \frac{1}{\lambda s + 1}$$

$$1 - \phi G = 1 - \frac{1}{\lambda s + 1} = \frac{\lambda s}{\lambda s + 1}$$

Here

$$K(s) = \frac{\phi(s)}{1 - \phi G(s)} = \frac{15.2s + 1}{3.24(\lambda s + 1)} \times \frac{\lambda s + 1}{\lambda s} = \frac{15.2s + 1}{3.24\lambda s}$$

$$= \frac{15.2}{3.24\lambda} + \frac{1}{3.24\lambda} s \leftarrow \text{This is your PI controller where } K_p = \frac{15.2}{3.24\lambda} \text{ and } K_I = \frac{1}{3.24\lambda}$$

Here λ becomes your tuning parameter (in fact the only tuning parameter). Just choose λ , and you have your PI gains.

Following similar procedure, the controller for tank 2 is developed as follows:

$$G_2(s) = \frac{K_{dc-2}}{T_2 s + 1} = \frac{1}{15.2s + 1} = \frac{X_2(s)}{X_1(s)}$$

and $\phi_0(s) = G_2^{-1}(s) = \frac{15.2s + 1}{1}$ — This is improper

$$\phi(s) = \phi_0(s) f(s) = \frac{15.2s + 1}{\lambda_2 s + 1}$$

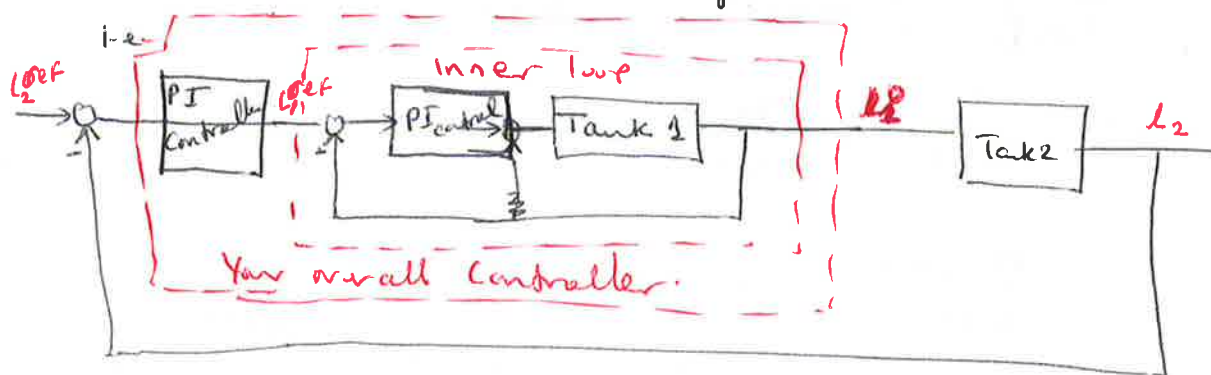
and then

$$K(s) = \frac{\phi(s)}{1 - G(s)\phi(s)} = \frac{15.2s + 1}{\lambda_2 s + 1} \times \frac{\lambda_2 s + 1}{\lambda_2 s} = \frac{15.2s + 1}{\lambda_2 s}$$

$$= \frac{15.2}{\lambda_2} + \frac{1}{\lambda_2 s} \quad \text{Here } K_p = \frac{15.2}{\lambda_2} \text{ and } K_I = \frac{1}{\lambda_2}$$

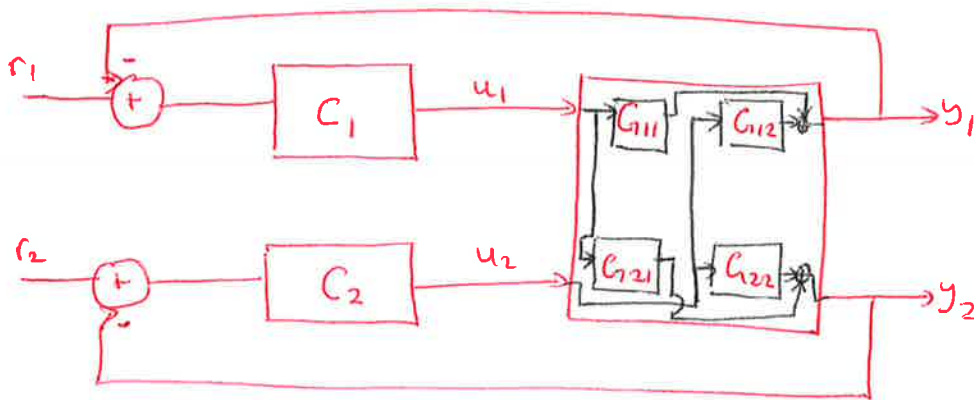
So again, choose any λ_2 , and you have your PI gains.

NOTE: To implement the controller for tank 2, you must embed the controller for tank 1 into your ^{overall} control implementation. Note that the set-point for tank 2 control is determined by tank 1 control.



Decentralised Control Design for Coupled-Tank sys

$$G = \begin{bmatrix} \frac{10.2}{12.44s + 1} & \frac{5.736}{(12.44s + 1)(6.35s + 1)} \\ \frac{6.787}{(14.72s + 1)(3.42s + 1)} & \frac{12.07}{14.72s + 1} \end{bmatrix}$$



where $C_i = K_i \left(1 + \frac{1}{T_{ci}s}\right)$ $i=1, 2$ or $C_i = K_{pi} + \frac{K_{Ii}}{s}$

We design C_1 based on $G_{11}(s)$ using the internal model ~~principle~~ ^{control} techniques.

$$\phi_{01} = G_{11}^{-1}(s) = \frac{12.44s + 1}{10.2}$$

Augmenting with a first order filter, we have

$$\phi_1 = G_{11}^{-1}(s) f(s) = \frac{12.44s + 1}{10.2} \times \frac{1}{\lambda_1 s + 1} = \frac{12.44s + 1}{10.2 \lambda_1 s + 10.2}$$

But $C_1 = \frac{\phi_1}{1 - \phi_1 G_{11}}$ where $\phi_1 G_{11} = \frac{1}{\lambda_1 s + 1}$ and $1 - \phi_1 G_{11} = \frac{\lambda_1 s}{\lambda_1 s + 1}$

Here $C_1 = \frac{12.44s + 1}{10.2 (\lambda_1 s + 1)} \times \frac{\lambda_1 s + 1}{\lambda_1 s} = \frac{12.44s + 1}{10.2 \lambda_1 s} = \frac{1.2197}{\lambda_1} + \frac{0.0981}{\lambda_1 s}$

Following similar techniques, we develop C_2 as follows:

$$\phi_{02} = G_{22}^{-1}(s) = \frac{14.72s + 1}{12.07}$$

Augmenting with a first-order filter, we have

$$\phi_2 = \phi_{02} f_2 = \frac{14.72s + 1}{12.07} \times \frac{1}{\lambda_2 s + 1} = \frac{14.72s + 1}{12.07(\lambda_2 s + 1)}$$

$$\text{and } C_2 = \frac{\phi_2}{1 - \phi_2 G_{22}} = \frac{14.72s + 1}{12.07(\lambda_2 s + 1)} \times \frac{\lambda_2 s + 1}{\lambda_2 s} = \frac{14.72s + 1}{12.07 \lambda_2 s} = \frac{1.2197}{\lambda_2} + \frac{0.0829}{\lambda_2 s}$$

Decoupling Controller Based on Internal model Control

$$G(s) = \begin{bmatrix} G_{11}(s) & G_{12}(s) \\ G_{21}(s) & G_{22}(s) \end{bmatrix} = \begin{bmatrix} \frac{10.2}{12.44s+1} & \frac{5.736}{(12.44s+1)(6.35s+1)} \\ \frac{6.787}{(14.72s+1)(3.42s+1)} & \frac{12.07}{14.72s+1} \end{bmatrix}$$

$$G^{-1}(s) = \begin{bmatrix} G_{22}(s) & -G_{12}(s) \\ -G_{21}(s) & G_{11}(s) \end{bmatrix} / \det(G(s))$$

$$\det G(s) = \left(\frac{10.2}{12.44s+1} \right) \left(\frac{12.07}{14.72s+1} \right) - \left(\frac{6.787}{(14.72s+1)(3.42s+1)} \right) \left(\frac{5.736}{(12.44s+1)(6.35s+1)} \right)$$

$$= \frac{(10.2)(12.07)(3.42s+1)(6.35s+1) - (6.787)(5.736)}{(12.44s+1)(14.72s+1)(3.42s+1)(6.35s+1)}$$

$$= \frac{2674s^2 + 1203s + 84.18}{(12.44s+1)(14.72s+1)(3.42s+1)(6.35s+1)}$$

$$= \frac{84.18(2.7533s+1)(11.534s+1)}{(12.44s+1)(14.72s+1)(3.42s+1)(6.35s+1)}$$

$$G^{-1}(s) = \begin{bmatrix} \frac{12.07}{14.72s+1} & -\frac{5.736}{(12.44s+1)(6.35s+1)} \\ -\frac{6.787}{(14.72s+1)(3.42s+1)} & \frac{10.2}{12.44s+1} \end{bmatrix} \times \frac{(12.44s+1)(14.72s+1)(3.42s+1)(6.35s+1)}{84.18(2.7533s+1)(11.534s+1)}$$

$$= \frac{0.0119}{(2.7533s+1)(11.534s+1)} \begin{bmatrix} 12.07(12.44s+1)(3.42s+1)(6.35s+1) & -5.736(14.72s+1)(3.42s+1) \\ -6.787(12.44s+1)(6.35s+1) & 10.2(14.72s+1)(3.42s+1)(6.35s+1) \end{bmatrix}$$

$$G^{-1}(s) = \begin{bmatrix} 0.1436 \frac{(12.44s+1)(3.42s+1)(6.35s+1)}{(2.7533s+1)(11.534s+1)} & -0.0683 \frac{(14.72s+1)(3.42s+1)}{(2.7533s+1)(11.534s+1)} \\ -0.0808 \frac{(12.44s+1)(6.35s+1)}{(2.7533s+1)(11.534s+1)} & 0.1214 \frac{(14.72s+1)(3.42s+1)(6.35s+1)}{(2.7533s+1)(11.534s+1)} \end{bmatrix}$$

To make ϕ bi-proper, we choose ϕ as

$$\phi = G^{-1}(s) f(s) \quad \text{where} \quad f(s) = \begin{bmatrix} \frac{1}{\lambda_1 s + 1} & 0 \\ 0 & \frac{1}{\lambda_2 s + 1} \end{bmatrix}$$

$$\phi = \begin{bmatrix} 0.1436 \frac{(12.44s+1)(3.42s+1)(6.35s+1)}{(\lambda_1 s + 1)(2.7533s+1)(11.534s+1)} & -0.0683 \frac{(14.72s+1)(3.42s+1)}{(\lambda_2 s + 1)(2.7533s+1)(11.534s+1)} \\ -0.0808 \frac{(12.44s+1)(6.35s+1)}{(\lambda_1 s + 1)(2.7533s+1)(11.534s+1)} & 0.1214 \frac{(14.72s+1)(3.42s+1)(6.35s+1)}{(\lambda_2 s + 1)(2.7533s+1)(11.534s+1)} \end{bmatrix}$$

Using the connection between internal model controller ϕ and the feedback controller $K(s)$ which is

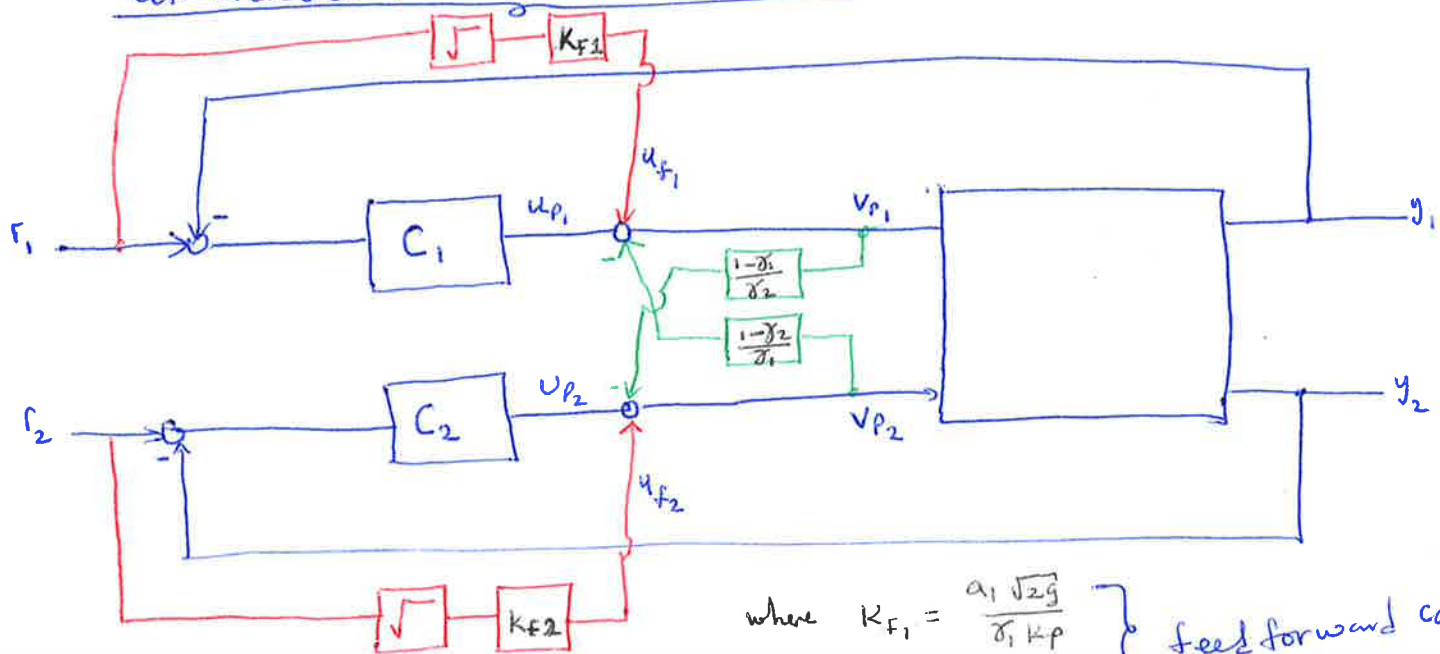
$$K(s) = \phi(s)(1 - G\phi)^{-1} = (1 - \phi G)^{-1} \phi$$

$$\text{with } 1 - G\phi = \begin{bmatrix} I & 0 \\ 0 & I \end{bmatrix} - \begin{bmatrix} \frac{1}{\lambda_1 s + 1} & 0 \\ 0 & \frac{1}{\lambda_2 s + 1} \end{bmatrix} = \begin{bmatrix} \frac{\lambda_1 s}{\lambda_1 s + 1} & 0 \\ 0 & \frac{\lambda_2 s}{\lambda_2 s + 1} \end{bmatrix}$$

$$\text{and } (1 - G\phi)^{-1} = \begin{bmatrix} \frac{\lambda_1 s + 1}{\lambda_1 s} & 0 \\ 0 & \frac{\lambda_2 s + 1}{\lambda_2 s} \end{bmatrix}$$

$$\text{Hence } K(s) = \phi(s)(1 - G\phi)^{-1} = \begin{bmatrix} 0.1436 \frac{(12.44s+1)(3.42s+1)(6.35s+1)}{\lambda_1 s (2.7533s+1)(11.534s+1)} & -0.0683 \frac{(14.72s+1)(3.42s+1)}{\lambda_2 s (2.7533s+1)(11.534s+1)} \\ -0.0808 \frac{(12.44s+1)(6.35s+1)}{\lambda_1 s (2.7533s+1)(11.534s+1)} & 0.1214 \frac{(14.72s+1)(3.42s+1)(6.35s+1)}{\lambda_2 s (2.7533s+1)(11.534s+1)} \end{bmatrix}$$

Multivariable PI Control with Feedback control



where $K_{F1} = \frac{a_1 \sqrt{2g}}{\gamma_1 K_P}$
 $K_{F2} = \frac{a_2 \sqrt{2g}}{\gamma_2 K_P}$ } feedforward control gains

where $C_1 = K_1 \left(1 + \frac{1}{T_{c1}s}\right)$
 $C_2 = K_2 \left(1 + \frac{1}{T_{c2}s}\right)$ } feedback PI controllers

Recall from the equilibrium analysis of the open-loop nonlinear plant

that

$$V_{P1}^0 = \frac{a_1}{\gamma_1 K_P} \sqrt{2g L_1^0} - \left(\frac{1-\gamma_2}{\gamma_1}\right) V_{P2}^0$$

and $V_{P2}^0 = \frac{a_3}{(1-\gamma_2) K_P} \sqrt{2g L_3^0}$

$$V_{P2}^0 = \frac{a_2}{\gamma_2 K_P} \sqrt{2g L_2^0} - \left(\frac{1-\gamma_1}{\gamma_2}\right) V_{P1}^0$$

$$V_{P1}^0 = \frac{a_4}{(1-\gamma_1) K_P} \sqrt{2g L_4^0}$$

From the control configuration above, equilibrium condition implies

$$r_1^0 - y_1^0 = 0 \Rightarrow u_{P1}^0 = 0 \Rightarrow v_{P1}^0 = u_{F1}^0 + \frac{1-\gamma_2}{\gamma_1} V_{P2}^0$$

$$r_2^0 - y_2^0 = 0 \Rightarrow u_{P2}^0 = 0 \Rightarrow v_{P2}^0 = u_{F2}^0 - \frac{1-\gamma_1}{\gamma_2} V_{P1}^0$$

Substituting into the equilibrium condition of the open-loop plant gives

$$V_{P1}^0 = \frac{a_1}{\gamma_1 K_P} \sqrt{2g L_1^0} - \left(\frac{1-\gamma_2}{\gamma_1}\right) \left(u_{F2}^0 - \frac{1-\gamma_1}{\gamma_2} V_{P1}^0\right)$$

$$V_{P1}^0 = K_{F1} \sqrt{L_1^0} - \frac{1-\gamma_2}{\gamma_1} V_{P2}^0$$

and

$$V_{P2}^0 = K_{F2} \sqrt{L_2^0} - \frac{1-\gamma_1}{\gamma_2} V_{P1}^0$$

Comparing the two equilibrium conditions gives

$$K_{F_1} = \frac{a_1}{\gamma_1 K_P} \sqrt{2g} \quad \text{and}$$

$$K_{F_2} = \frac{a_2}{\gamma_2 K_P} \sqrt{2g}$$

Decentralised PI Control Design

We already designed C_1 and C_2 based on the internal model control design technique. We used the linearized plant and ignored the couplings.

Centralised Steady-State Decoupler

To design a centralised controller which decouples the linearized plant at steady state, we ~~adopt~~ ^{follow} the ~~following~~ ^{design} technique below:

$$C = \begin{bmatrix} C_1 & \\ & C_2 \end{bmatrix} K_s^{-1} \quad \text{where } K_s \text{ is the steady-state gain of the linearized plant.}$$

$$\text{i.e. } K_s = \lim_{s \rightarrow 0} G(s) = -CA^{-1}B$$

Model Predictive Control formulation for Quadraple 1

Bank System

1. Linearized plant model (Discrete)

$$x(t+1) = \overset{n \times n}{A} x(t) + \overset{n \times m}{B} u(t)$$

$$y(t) = \overset{m \times n}{C} x(t)$$

2. The observer To capture the mismatch between the actual plant and the linearized plant in steady state.

first augments the linear plant with internal model of the disturbance — constant disturbance

$$x(t+1) = \overset{n \times n}{A} x(t) + \overset{n \times m}{B} u(t) + \overset{n \times 1}{d}(t)$$

$$d(t+1) = \overset{m \times 1}{d}(t)$$

$$y(t) = \overset{m \times n}{C} x(t) + \overset{m \times 1}{d}(t)$$

Integrator — adding integral action.
Disturbance assumed to be at the plant output.

The augmented system becomes

$$\begin{bmatrix} x(t+1) \\ d(t+1) \end{bmatrix} = \begin{bmatrix} \overset{n \times n}{A} & \overset{n \times m}{0} \\ \overset{m \times n}{0} & \overset{m \times m}{I} \end{bmatrix} \begin{bmatrix} x(t) \\ d(t) \end{bmatrix} + \begin{bmatrix} \overset{n \times m}{B} \\ \overset{m \times m}{0} \end{bmatrix} u(t)$$

$$y(t) = \begin{bmatrix} \overset{m \times n}{C} & \overset{m \times m}{I} \end{bmatrix} \begin{bmatrix} x(t) \\ d(t) \end{bmatrix}$$

We consider the state and disturbance estimator

$$\begin{bmatrix} \hat{x}(t+1) \\ \hat{d}(t+1) \end{bmatrix} = \begin{bmatrix} \overset{n \times n}{A} & \overset{n \times m}{0} \\ \overset{m \times n}{0} & \overset{m \times m}{I} \end{bmatrix} \begin{bmatrix} \hat{x}(t) \\ \hat{d}(t) \end{bmatrix} + \begin{bmatrix} \overset{n \times m}{B} \\ \overset{m \times m}{0} \end{bmatrix} u(t) + \begin{bmatrix} L_x \\ L_d \end{bmatrix} (-y_m(t) + \underbrace{C \hat{x}(t) + \hat{d}(t)}_{\substack{\text{output of the} \\ \text{estimator/observer}}})$$

Actual plant output (measured)

Choose L_x and L_d such that the estimator is "stable".

$$\therefore \begin{bmatrix} \hat{x}(t+1) \\ \hat{d}(t+1) \end{bmatrix} = \begin{bmatrix} A + L_x C & L_x \\ L_d C & I + L_d \end{bmatrix} \begin{bmatrix} \hat{x}(t) \\ \hat{d}(t) \end{bmatrix} + \begin{bmatrix} B \\ 0 \end{bmatrix} u(t) - \begin{bmatrix} L_x \\ L_d \end{bmatrix} y_m(t)$$

No poles at (1,0)

3. Steady State Analysis

At steady state, we have

$$\begin{aligned} \hat{x}_\infty &= A \hat{x}_\infty + B u_\infty \\ y_{m,\infty} &= C \hat{x}_\infty + \hat{d}_\infty \end{aligned} \quad \sim \quad \begin{bmatrix} n \times n & n \times m \\ m \times n & 0 \end{bmatrix} \begin{bmatrix} \hat{x}_\infty \\ u_\infty \end{bmatrix} = \begin{bmatrix} 0 \\ y_{m,\infty} - \hat{d}_\infty \end{bmatrix}$$

$$\omega \quad \begin{bmatrix} A-I & B \\ C & 0 \end{bmatrix} \begin{bmatrix} \hat{x}_\infty \\ u_\infty \end{bmatrix} = \begin{bmatrix} 0 \\ r_\infty - \hat{d}_\infty \end{bmatrix}$$

reference.

The solution of $\begin{bmatrix} \hat{x}_\infty \\ u_\infty \end{bmatrix}$ can be parameterized by $r_\infty - \hat{d}_\infty$. For solution to exist, the above matrix $\begin{bmatrix} A-I & B \\ C & 0 \end{bmatrix}$ must be full row-ranked, i.e. (measured output) the rows of C must be less ^{or equal to} the columns of B i.e. (no of control input).

4. The MPC design

The MPC design is as follows:

This is satisfied for our case, since we have two measured outputs and two control inputs.

$$\min_{u_0, \dots, u_{N-1}} \|x_N - \bar{x}_t\|_P^2 + \sum_{k=0}^{N-1} \|x_k - \bar{x}_t\|_\phi^2 + \|u_k - \bar{u}_t\|_R^2$$

Subject to

$$\begin{aligned} \underline{x} &\leq x_k \leq \bar{x} \quad k=0, \dots, N && \text{--- Constraints on states} \\ \underline{u} &\leq u_k \leq \bar{u} \quad k=0, \dots, N && \text{--- Constraints on inputs} \\ x_{k+1} &= A x_k + B u_k + d_k, \quad k=0, \dots, N-1 && \text{prediction model for state and disturbance.} \\ d_{k+1} &= d_k, \quad k=0, \dots, N-1 \\ x_0 &= \hat{x}(t) && \text{--- initial state} \\ d_0 &= \hat{d}(t) && \text{--- initial disturbance} \end{aligned}$$

with \bar{u}_t and \bar{x}_t given by

$$\begin{bmatrix} A-I & B \\ C & 0 \end{bmatrix} \begin{bmatrix} \bar{x}_t \\ \bar{u}_t \end{bmatrix} = \begin{bmatrix} 0 \\ r(t) - \hat{d}(t) \end{bmatrix}$$

} Steady target state

where $\|x\|_M^2 \triangleq x^T M x$, $\phi \geq 0$, $R > 0$ and P satisfies the Riccati equation

$$P = A^T P A - (A^T P B)(B^T P B + R)^{-1}(B^T P A) + \phi$$

Suppose

$$\begin{bmatrix} A-I & B \\ C & 0 \end{bmatrix}^{-1} = \begin{bmatrix} * & M \\ * & N \end{bmatrix} \quad \text{so} \quad \begin{bmatrix} A-I & B \\ C & 0 \end{bmatrix} \begin{bmatrix} \bar{x}_t \\ \bar{u}_t \end{bmatrix} = \begin{bmatrix} 0 \\ r(t) - \hat{J}(t) \end{bmatrix} \quad \text{can be}$$

expressed as

$$\begin{bmatrix} \bar{x}_t \\ \bar{u}_t \end{bmatrix} = \begin{bmatrix} M \\ N \end{bmatrix} (r(t) - \hat{J}(t))$$

The objective function

So the objective function can be re-written as:

$$V_N = (x_N - Mw)^T P (x_N - Mw) + (x_0 - Mw)^T \phi (x_0 - Mw) + (x_1 - Mw)^T \phi (x_1 - Mw) + \dots + (x_{N-1} - Mw)^T \phi (x_{N-1} - Mw) + (u_0 - \hat{N}w)^T R (u_0 - \hat{N}w) + \dots + (u_{N-1} - \hat{N}w)^T R (u_{N-1} - \hat{N}w)$$

(n+m)N — Prediction Horizon

$$\begin{bmatrix} u_0^T & x_1^T & u_1^T & \dots & u_{N-1}^T & x_N^T \end{bmatrix} \begin{bmatrix} R & \phi & R & \dots & R & P \\ & & & & & \\ & & & & & \\ & & & & & \\ & & & & & \end{bmatrix} \begin{bmatrix} u_0 \\ x_1 \\ u_1 \\ \vdots \\ u_{N-1} \\ x_N \end{bmatrix}$$

$$-2 \begin{bmatrix} u_0^T & x_1^T & u_1^T & \dots & u_{N-1}^T & x_N^T \end{bmatrix} \begin{bmatrix} R \hat{N} w \\ \phi M w \\ R \hat{N} w \\ \vdots \\ R \hat{N} w \\ P M w \end{bmatrix} + (x_0 - Mw)^T \phi (x_0 - Mw) + (N-3) w^T \hat{N}^T \phi M w + w^T \hat{N}^T P M w + N w^T \hat{N}^T R \hat{N} w$$

Constant terms

Define

$$z = \begin{bmatrix} u_0 \\ x_1 \\ u_1 \\ \vdots \\ u_{N-1} \\ x_N \end{bmatrix} \quad H = \begin{bmatrix} R & \phi & R & \dots & R & P \\ & & & & & \\ & & & & & \\ & & & & & \\ & & & & & \end{bmatrix} \quad \text{and} \quad F = \begin{bmatrix} \hat{N} \\ M \\ \hat{N} \\ \vdots \\ \hat{N} \\ M \end{bmatrix} \begin{matrix} f_0 \\ \\ \\ \\ \\ \end{matrix}$$

we have the objective as

$$z^T H z - 2 z^T F w + \text{constant terms.}$$

where $F = H F_0$ and $w = r(t) - \hat{d}(t)$
 |
 reference input estimated disturbance (from observer)

The Prediction

$$\begin{aligned} x_{k+1} &= A x_k + B u_k + d_k \\ d_{k+1} &= d_k \end{aligned} \quad \} \quad k=0, \dots, N$$

$k=0$: $x_1 = A x_0 + B u_0 + d_0$
 $d_1 = d_0$
 $\Rightarrow x_1 - B u_0 - d_0 = A x_0$
 or
 $x_1 - B u_0 = A x_0 + d_0$

$k=1$: $x_2 = A x_1 + B u_1 + d_1$
 $d_2 = d_1$
 $\Rightarrow x_2 - A x_1 - B u_1 = d_0$

$k=2$: $x_3 = A x_2 + B u_2 + d_2$
 $d_3 = d_2$
 $\Rightarrow x_3 - A x_2 - B u_2 = d_0$

$k=N$: $x_{N+1} = A x_N + B u_N + d_N, d_{N+1} = d_N$

$$\begin{aligned} & \underbrace{\begin{bmatrix} -B & I & & & \\ 0 & -A & -B & I & \\ 0 & 0 & 0 & -A & -B & I \\ & & \ddots & & & \\ 0 & & & -A & -B & I \end{bmatrix}}_{(n+m)N \times (n+m)N} \underbrace{\begin{bmatrix} u_0 \\ x_1 \\ u_1 \\ \vdots \\ u_{N-1} \\ x_N \end{bmatrix}}_{(n+m)N \times 1} = \underbrace{\begin{bmatrix} A & I \\ 0 & I \\ 0 & I \\ \vdots & \\ 0 & I \end{bmatrix}}_{nN \times (n+m)} \underbrace{\begin{bmatrix} x_0 \\ d_0 \end{bmatrix}}_{e} \end{aligned}$$

$x_N - A x_{N-1} - B u_{N-1} = d_0$

In compact form, we have

$$E z = e$$

where

$$e = F_1 \hat{x}(t) + F_2 \hat{d}(t)$$

$$F_1 = \begin{bmatrix} A \\ 0 \\ 0 \\ \vdots \\ 0 \end{bmatrix} \quad F_2 = \begin{bmatrix} I \\ I \\ I \\ \vdots \\ I \end{bmatrix}$$

estimated state
 estimated disturbance
 from the observer.

Constraints

(5)

$$\underline{x} \leq x_k \leq \bar{x}$$

$$k=0, \dots, N$$

$$\underline{u} \leq u_k \leq \bar{u}$$

$$\begin{matrix} (n+m)N \times 1 \\ \begin{bmatrix} \underline{u} \\ \underline{x} \\ \underline{u} \\ \vdots \\ \underline{u} \\ \underline{x} \end{bmatrix} \end{matrix} \leq \begin{matrix} (n+m)N \times 1 \\ \begin{bmatrix} u_0 \\ x_1 \\ u_1 \\ \vdots \\ u_{N-1} \\ x_N \end{bmatrix} \end{matrix} \leq \begin{matrix} (n+m)N \times 1 \\ \begin{bmatrix} \bar{u}_0 \\ \bar{x} \\ \bar{u} \\ \vdots \\ \bar{u} \\ \bar{x} \end{bmatrix} \end{matrix}$$

In Compact form, we have

$$\underline{z} \leq z \leq \bar{z}$$

Lower bound upper bound

Putting all together, we have the following Quadratic Program:

$$\min_z \frac{1}{2} z^T H z - z^T q(t)$$

$$\text{Subject to } E z = e(t)$$

$$\underline{z} \leq z \leq \bar{z}$$

$$\text{where } q(t) = F(r(t) - \hat{J}(t)) = H F_0 (r(t) - \hat{J}(t))$$

$$\text{and } e(t) = F_1 \hat{x}(t) + F_2 \hat{J}(t)$$

NOTES:

1. We only apply the first element of the ~~decision~~ ^{decision} variable z i.e. u_0
2. Recall that we linearized our plant about operating points (x_0^0, u_0^0) , so

$$x = x_0 + x^0 \quad \text{and}$$

Actual state estimated state by observer operating point

$$u = u_0 + u^0$$

operating point computed control effort by MPC

We must apply u to the actual plant

So r must be adjusted by r^0 before passing into the MPC i.e.

$$r_0 = r - r^0$$

Actual reference carries point operating point

What is given to MPC: Actual reference

3. The reformulated QP can be solved either in digital form (Halevy & Jeff) or in continuous time (Terrence).
4. The observer/estimator has to be implemented.

Observer implementation

In Discrete:

$$\begin{bmatrix} \hat{x}(t+1) \\ \hat{d}(t+1) \end{bmatrix} = \begin{bmatrix} A & 0 \\ 0 & I \end{bmatrix} \begin{bmatrix} \hat{x}(t) \\ \hat{d}(t) \end{bmatrix} + \begin{bmatrix} B \\ 0 \end{bmatrix} u(t) + \begin{bmatrix} L_x \\ L_d \end{bmatrix} (c \hat{x}(t) + \hat{d}(t) - y_m(t))$$

This has to be computed using "kalman" a "plane" in Matlab.
 same control applied to the actual plant
 As before, this is the u_0 computed by the MPC.

Actual plant output (measured height).
 As before dictated by the operating point.

In Continuous time

Linearised plant

$$\dot{x} = Ax + Bu$$

$$y = Cx$$

Augment the plant with a disturbance model as

$$\dot{x} = Ax + Bu$$

$$\dot{d} = d$$

$$y = Cx + d$$

Integrator - adding integral action

Such that we have:

$$\begin{bmatrix} \dot{x} \\ \dot{d} \end{bmatrix} = \begin{bmatrix} A & 0 \\ 0 & I \end{bmatrix} \begin{bmatrix} x \\ d \end{bmatrix} + \begin{bmatrix} B \\ 0 \end{bmatrix} u$$

$$y = [C \quad I] \begin{bmatrix} x \\ d \end{bmatrix}$$

The observer or state estimator becomes

$$\begin{bmatrix} \dot{\hat{x}} \\ \dot{\hat{d}} \end{bmatrix} = \begin{bmatrix} A & 0 \\ 0 & I \end{bmatrix} \begin{bmatrix} \hat{x} \\ \hat{d} \end{bmatrix} + \begin{bmatrix} B \\ 0 \end{bmatrix} u + \begin{bmatrix} L_x \\ L_d \end{bmatrix} [\hat{y} - y]$$

Output of the estimator

where $\hat{y} = C\hat{x} + \hat{d}$
 output of the actual plant adjusted for the operating condition.

observer gains to be computed such that the observer is stable.

Note: we have abuse the notation for the system parameters A, B, C .
 Here they are the actual system parameters of the plant.

For the discrete version, (A, B, C) has to be discretised using appropriate sampling time to obtain new parameters.

MPC Design for the Case without state constraints:

Suppose there are no state constraints so that the MPC problem becomes:

$$\min_{u_0, \dots, u_{N-1}} \|x_N - \bar{x}_t\|_p^2 + \sum_{k=0}^{N-1} \|x_k - \bar{x}_t\|_q^2 + \|u_k - \bar{u}_t\|_R^2$$

Subject to:

$$\underline{u} \leq u_k \leq \bar{u}, \quad k=0, \dots, N$$

$$x_{k+1} = Ax_k + Bu_k + d_k, \quad k=0, \dots, N$$

$$d_{k+1} = d_k, \quad k=0, \dots, N$$

$$x_0 = \hat{x}(t)$$

$$d_0 = \hat{d}(t)$$

with \bar{u}_t and \bar{x}_t given by

$$\begin{bmatrix} A-1 & B \\ C & 0 \end{bmatrix} \begin{bmatrix} \bar{x}_t \\ \bar{u}_t \end{bmatrix} = \begin{bmatrix} 0 \\ r(t) - \hat{d}(t) \end{bmatrix}$$

As before, we parameterize the target solution in terms of the

$$\text{input } r(t) - \hat{d}(t) \approx \begin{bmatrix} \bar{x}_t \\ \bar{u}_t \end{bmatrix} = \begin{bmatrix} M \\ \hat{N} \end{bmatrix} (r(t) - \hat{d}(t))$$

The prediction:

$$k=0 \quad x_1 = Ax_0 + Bu_0 + d_0, \quad d_1 = d_0$$

$$\begin{aligned} k=1 \quad x_2 &= Ax_1 + Bu_1 + d_1, \quad d_2 = d_1 \\ &= A^2x_0 + ABu_0 + Ad_0 + Bu_1 + d_0 \end{aligned}$$

$$\begin{aligned} k=2 \quad x_3 &= Ax_2 + Bu_2 + d_2, \quad d_3 = d_2 \\ &= A^3x_0 + A^2Bu_0 + A^2d_0 + ABu_1 + Ad_0 + Bu_2 + d_0 \end{aligned}$$

$$k=3 \quad x_4 = Ax_3 + Bu_3 + d_3, \quad d_4 = d_3$$

$$x_4 = A^4 x_0 + A^3 B u_0 + A^3 d_0 + A^2 B u_1 + A^2 d_0 + A B u_2 + A d_0 + B u_3 + d_0$$

$$\vdots$$

$$x_N = A^N x_0 + (A^{N-1} + A^{N-2} + \dots + A + I) d_0 + A^{N-1} B u_0 + A^{N-2} B u_1 + \dots + A B u_{N-2} + B u_{N-1}$$

So

$$\begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_N \end{bmatrix} = \begin{bmatrix} A \\ A^2 \\ A^3 \\ \vdots \\ A^N \end{bmatrix} x_0 + \begin{bmatrix} I \\ A+I \\ A^2+A+I \\ \vdots \\ A^{N-1} + \dots + A + I \end{bmatrix} d_0 + \begin{bmatrix} B & & & \\ AB & B & & \\ A^2 B & AB & B & \\ \vdots & \vdots & \vdots & \ddots \\ A^{N-1} B & A^{N-2} B & \dots & AB & B \end{bmatrix} \begin{bmatrix} u_0 \\ u_1 \\ u_2 \\ \vdots \\ u_{N-1} \end{bmatrix}$$

Define

$$X = \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_N \end{bmatrix}, \quad U = \begin{bmatrix} u_0 \\ u_1 \\ u_2 \\ \vdots \\ u_{N-1} \end{bmatrix}, \quad \Phi = \begin{bmatrix} A \\ A^2 \\ A^3 \\ \vdots \\ A^N \end{bmatrix}, \quad \Psi = \begin{bmatrix} I \\ A+I \\ A^2+A+I \\ \vdots \\ A^{N-1} + \dots + A + I \end{bmatrix}, \quad \Lambda = \begin{bmatrix} B & & & \\ AB & B & & \\ A^2 B & AB & B & \\ \vdots & \vdots & \vdots & \ddots \\ A^{N-1} B & A^{N-2} B & \dots & AB & B \end{bmatrix}$$

So in Compact form

$$X = \Phi x_0 + \Psi d_0 + \Lambda U$$

The constraints

$$\underline{u} \leq u_k \leq \bar{u} \quad k=0, \dots, N-1$$

$$\begin{bmatrix} \underline{u} \\ \underline{u} \\ \underline{u} \\ \vdots \\ \underline{u} \end{bmatrix} \leq \begin{bmatrix} u_0 \\ u_1 \\ u_2 \\ \vdots \\ u_{N-1} \end{bmatrix} \leq \begin{bmatrix} \bar{u} \\ \bar{u} \\ \bar{u} \\ \vdots \\ \bar{u} \end{bmatrix}$$

This can be rewritten compactly as

$$\underline{u} \leq U \leq \bar{u}$$

Objective Function

$$\sum_{k=0}^{N-1} \|x_k - \bar{x}_k\|_\varphi^2 + \|x_N - \bar{x}_N\|_\rho^2$$

$$= (x_0 - M\omega)^T \varphi (x_0 - M\omega) + (x_1 - M\omega)^T \varphi (x_1 - M\omega) + \dots + (x_{N-1} - M\omega)^T \varphi (x_{N-1} - M\omega) \\ + (x_N - M\omega)^T \rho (x_N - M\omega)$$

$$= [x_1^T \ x_2^T \ \dots \ x_{N-1}^T \ x_N^T] \begin{bmatrix} \varphi & & & \\ & \varphi & & \\ & & \ddots & \\ & & & \varphi & \\ & & & & \rho \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_{N-1} \\ x_N \end{bmatrix}$$

$$- 2 [x_1^T \ x_2^T \ \dots \ x_{N-1}^T \ x_N^T] \begin{bmatrix} \varphi & & & \\ & \varphi & & \\ & & \ddots & \\ & & & \varphi & \\ & & & & \rho \end{bmatrix} \begin{bmatrix} M \\ M \\ \vdots \\ M \\ M \end{bmatrix} \omega$$

$$+ x_0^T \varphi x_0 + \underbrace{w^T M^T \varphi M w + w^T M^T \rho M w}_{\text{constants}}$$

Compactly

$$= X^T \tilde{\Phi} X - 2 X^T \tilde{\Phi} \tilde{M} w + \text{constant terms.}$$

Also $\sum_{k=0}^{N-1} \|u_k - \bar{u}_k\|_R^2 = (u_0 - \hat{N}w)^T R (u_0 - \hat{N}w) + \dots + (u_{N-1} - \hat{N}w)^T R (u_{N-1} - \hat{N}w)$

$$= [u_0^T \ u_1^T \ \dots \ u_{N-1}^T] \begin{bmatrix} R & & & \\ & R & & \\ & & \ddots & \\ & & & R \end{bmatrix} \begin{bmatrix} u_0 \\ u_1 \\ \vdots \\ u_{N-1} \end{bmatrix} - 2 [u_0^T \ u_1^T \ \dots \ u_{N-1}^T] \begin{bmatrix} R\hat{N} \\ R\hat{N} \\ \vdots \\ R\hat{N} \end{bmatrix} w$$

$$+ N w^T \hat{N}^T R \hat{N} w$$

$$= U^T \tilde{R} U - 2 U^T \tilde{R} \tilde{N} w + \text{constant term.}$$

Putting all together and substituting for

$$X = \phi x_0 + \psi d_0 + \Lambda U$$

The objective becomes

$$= X^T \tilde{\Phi} X - 2 X^T \tilde{\Phi} \tilde{M} w + U^T \tilde{R} U - 2 U^T \tilde{R} \tilde{N} w + \text{constant}$$

$$= (\phi x_0 + \psi d_0 + \Lambda U)^T \tilde{\Phi} (\phi x_0 + \psi d_0 + \Lambda U) - 2 (\phi x_0 + \psi d_0 + \Lambda U)^T \tilde{\Phi} \tilde{M} w$$

$$+ U^T \tilde{R} U - 2 U^T \tilde{R} \tilde{N} w + \text{constants}$$

$$= x_0^T \phi^T \tilde{\Phi} \phi x_0 + x_0^T \phi^T \tilde{\Phi} \psi d_0 + x_0^T \phi^T \tilde{\Phi} \Lambda U$$

$$+ d_0^T \psi^T \tilde{\Phi} \phi x_0 + d_0^T \psi^T \tilde{\Phi} \psi d_0 + d_0^T \psi^T \tilde{\Phi} \Lambda U$$

$$+ U^T \Lambda^T \tilde{\Phi} \phi x_0 + U^T \Lambda^T \tilde{\Phi} \psi d_0 + U^T \Lambda^T \tilde{\Phi} \Lambda U$$

$$- 2 x_0^T \phi^T \tilde{\Phi} \tilde{M} w - 2 d_0^T \psi^T \tilde{\Phi} \tilde{M} w - 2 U^T \Lambda^T \tilde{\Phi} \tilde{M} w - 2 U^T \tilde{R} \tilde{N} w$$

$$+ U^T \tilde{R} U + \text{constant}$$

$$= U^T (\Lambda^T \tilde{\Phi} \Lambda + \tilde{R}) U + 2 U^T \Lambda^T \tilde{\Phi} \phi x_0 + 2 U^T \Lambda^T \tilde{\Phi} \psi d_0$$

$$- 2 U^T \Lambda^T \tilde{\Phi} \tilde{M} w - 2 U^T \tilde{R} \tilde{N} w + \text{constant terms.}$$

$$= U^T (\Lambda^T \tilde{\Phi} \Lambda + \tilde{R}) U + 2 U^T \Lambda^T \tilde{\Phi} \phi x_0 + 2 U^T (\Lambda^T \tilde{\Phi} \psi + \tilde{\Phi} \tilde{M} + \tilde{R} \tilde{N}) d_0$$

$$- 2 U^T (\Lambda^T \tilde{\Phi} \tilde{M} + \tilde{R} \tilde{N}) r(t) + \text{constant terms.}$$

The MPC problem can then be compactly described as

$$\min_{u_0, u_1, \dots, u_{N-1}} U^T H U + 2 U^T (F_1 x_0 + F_2 d_0 + F_3 r) + \text{constant terms}$$

$$\text{subject to } \underline{U} \leq U \leq \bar{U}$$

Consider the problem

$$\min_z \frac{1}{2} z^T H z - z^T q(\theta)$$

Subject to $Ez = e(\theta)$

$$z \leq \bar{z} \leq \bar{z}$$

where

$$H = \begin{bmatrix} R & & & \\ & \varnothing & & \\ & & R & \\ & & & \ddots \\ & & & & R \\ & & & & & P \end{bmatrix}$$

$$E = \begin{bmatrix} -B & I & & & \\ 0 & -A & -B & I & \\ & 0 & -A & -B & I \\ & & & & 0 & -A & -B & I \end{bmatrix}$$

$$q(\theta) = F(r(\theta) - \hat{d}(\theta)) \text{ with } F = HF_0, F_0 = \begin{bmatrix} \hat{N} \\ M \\ \vdots \\ \hat{N} \\ M \end{bmatrix}$$

$$\text{and } e(\theta) = F_1 \hat{x}(\theta) + F_2 \hat{d}(\theta) \text{ with } F_1 = \begin{bmatrix} A \\ 0 \\ 0 \\ \vdots \\ 0 \end{bmatrix} \text{ and } F_2 = \begin{bmatrix} I \\ I \\ \vdots \\ I \end{bmatrix}$$

For problem with a single prediction horizon ($N=1$)

$$H = \begin{bmatrix} R & 0 \\ 0 & P \end{bmatrix}, E = \begin{bmatrix} -B & I \end{bmatrix}, F = \begin{bmatrix} R \hat{N} \\ P M \end{bmatrix}, F_1 = A, F_2 = I$$

$$q(\theta) = \begin{bmatrix} R \hat{N} \\ P M \end{bmatrix} [r(\theta) - \hat{d}(\theta)]$$

using the fixed point iteration algorithm + Advantage and Skibitz

$$z^{k+1} = \underset{\text{Project}}{\phi} (z^k - \alpha D^T (Hz^k - E^T \lambda^k - q(\theta)))$$

$$\lambda^{k+1} = \lambda^k + \omega \phi^T (e(\theta) - Ez^{k+1})$$

After Euler approximation:

$$z \dot{z} = \phi_z (z - D^T (Hz - E^T \lambda - q(\theta))) - z$$

$$\lambda \dot{\lambda} = \phi_\lambda (e(\theta) - Ez)$$

We can take advantage of the conditioning matrices D and ϕ to scale our signals

Suppose H is positive definite then we choose D as the diagonal elements of H . ϕ can be chosen as EE^T or $EH^{-1}E^T$ or I $EE^T = \begin{bmatrix} -B^T \\ I \end{bmatrix} \begin{bmatrix} -B \\ I \end{bmatrix} = BB^T + I$

