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```
clear
clc
```

Inputs

```
%All in standard SI units unless noted
OperatingVoltage = 40                                %V
TorqueConstant = 8.474E-3
    %N.m/A
VoltageConstant = 1125
    %RPM/V
ArmatureResistance = 0.072
    %Ohms
reqOutputSpeed = 575
    %RPM
reqOutputTorque = 55
    %N.m
Efficiency = 90/100
Duty = 2000
    %Hours

%Gears
PressureAngle = 20*pi/180
    %Rad
FaceWidth = [1.25, 1, 0.75, 0.5, 0.375, 0.25, 0.188, 0.125]
    %inch
DiametralPitch = [8, 10, 12, 16, 20, 24, 32, 48]
    %teeth/inch
k = 1
    %Teeth Depth (1 = full)
m1 = 5
    %Gear Ratio Stage 1
m2 = 4
    %Gear Ratio Stage 2

OperatingVoltage =

40
```

TorqueConstant =

0.0085

VoltageConstant =

1125

ArmatureResistance =

0.0720

reqOutputSpeed =

575

reqOutputTorque =

55

Efficiency =

0.9000

Duty =

2000

PressureAngle =

0.3491

FaceWidth =

Columns 1 through 7

1.2500	1.0000	0.7500	0.5000	0.3750	0.2500	0.1880
--------	--------	--------	--------	--------	--------	--------

Column 8

0.1250

DiametralPitch =

```

            8      10      12      16      20      24      32      48

k =

    1

m1 =

    5

m2 =

    4

```

Motor Torque-Speed

```

%point 1:
NoLoadSpeed = VoltageConstant*OperatingVoltage;
NoLoadTorque = 0;
%point 2:
StallTorque = TorqueConstant*OperatingVoltage/ArmatureResistance;
StallSpeed = 0;

%determine Torque-speed graph
points = [StallSpeed, StallTorque;
          NoLoadSpeed, NoLoadTorque];
polynomialDegree = length(points)-1;
MotorLine = polyfit(points(:,1),points(:,2),polynomialDegree);
MotorLineE = Efficiency.*MotorLine;

f1 = figure('Renderer', 'painters', 'Position', [10 10 1200 300])
subplot(1,3,1)
plot([StallSpeed:NoLoadSpeed], polyval(MotorLine,
[StallSpeed:NoLoadSpeed]))
hold on
title("Motor Torque-Speed")
xlabel("Motor Speed (rpm)")
ylabel("Motor Torque (N.m)")
hold off

f1 =

```

Figure (6) with properties:

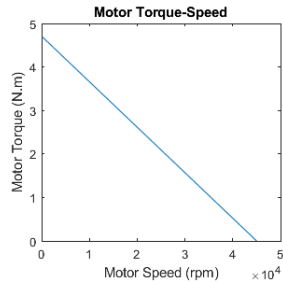
```

Number: 6
Name: ''
Color: [0.9400 0.9400 0.9400]

```

Position: [10 10 1200 300]
Units: 'pixels'

Use GET to show all properties



Gear Ratio Target (minimization)

```
syms r
eqn = MotorLineE(1)*reqOutputSpeed*r^2 + MotorLineE(2)*r ==
    reqOutputTorque;
soln = double(solve(eqn,r))

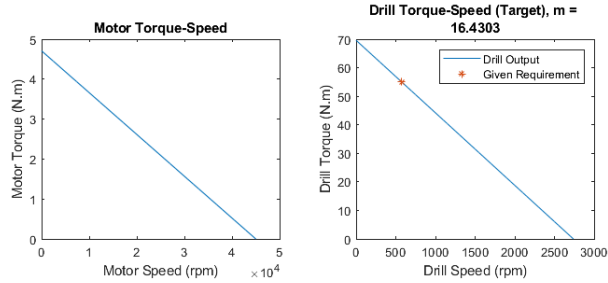
ratioTarget = soln(1);

TargetLine = [MotorLineE(1)*ratioTarget^2, MotorLineE(2)*ratioTarget];

subplot(1,3,2)
plot([StallSpeed/ratioTarget: NoLoadSpeed/ratioTarget],
    polyval(TargetLine,[StallSpeed/ratioTarget: NoLoadSpeed/
ratioTarget]))
hold on
plot(reqOutputSpeed, reqOutputTorque, '*')
title(["Drill Torque-Speed (Target), m = ", ratioTarget])
xlabel("Drill Speed (rpm)")
ylabel("Drill Torque (N.m)")
legend("Drill Output", "Given Requirement")
hold off

soln =

    16.4303
    61.8306
```



Gears

```
%pinions
P1 = ceil((2*k*(m1+(m1^2+(1-2*m1)*(sin(PressureAngle))^2)^0.5))/
((1+2*m1)*(sin(PressureAngle))^2));
P2 = ceil((2*k*(m2+(m2^2+(1-2*m2)*(sin(PressureAngle))^2)^0.5))/
((1+2*m2)*(sin(PressureAngle))^2));

%gear Limits
N1l = floor(((P1^2)*(sin(PressureAngle)^2)-4*k^2)/
(4*k-2*P1*sin(PressureAngle)^2));
N2l = floor(((P2^2)*(sin(PressureAngle)^2)-4*k^2)/
(4*k-2*P2*sin(PressureAngle)^2));

if(m1*P1 > N1l)
    N1 = N1l;    %Gear Ratio m is too large
    disp("Ratio m1 Too Large, increasing P1")
    P1 = P1+1;
    N1 = floor(m1*P1);
else
    N1 = floor(m1*P1);
end

if(m2*P2 > N2l);
    N2 = N2l;    %Gear Ratio m is too large
    disp("Ratio m2 Too Large, increasing P2")
    P2 = P2+1;
    N2 = floor(m2*P2);
else
    N2 = floor(m2*P2);
end

mlactual = N1/P1;
m2actual = N2/P2;
ratio = mlactual*m2actual;

RatioTable = table(m1, mlactual, P1, N1, m2, m2actual, P2, N2, ratio,
    ratioTarget)

RatioTable =

    1x10 table
```

<i>m1</i>	<i>m1actual</i>	<i>P1</i>	<i>N1</i>	<i>m2</i>	<i>m2actual</i>	<i>P2</i>	<i>N2</i>	<i>ratio</i>
<i>ratioTarget</i>								
5	5	16	80	4	4	16	64	20
16.43								

Gear Ratio (Overall)

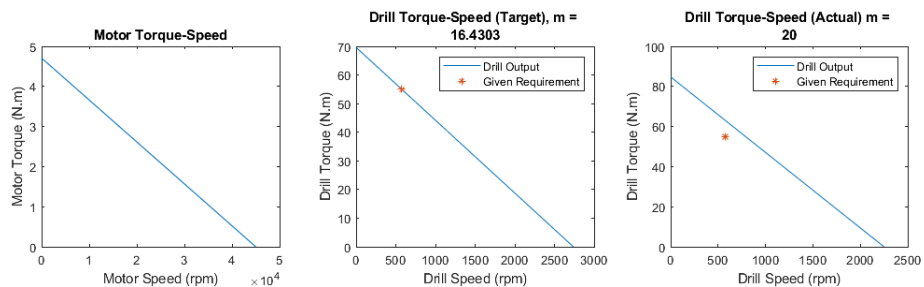
```

outputTorque = [0];
outputSpeed = [0];

RealLine = [MotorLineE(1)*ratio^2, MotorLineE(2)*ratio];

for x = [StallSpeed:NoLoadSpeed]
    outputSpeed = [outputSpeed, x/ratio];
    outputTorque = [outputTorque, polyval(RealLine,x/ratio)];
end
outputSpeed = outputSpeed(2:end);
outputTorque = outputTorque(2:end);
subplot(1,3,3)
plot(outputSpeed, outputTorque)
hold on
plot(reqOutputSpeed,reqOutputTorque,'*')
title(["Drill Torque-Speed (Actual) m = ", ratio])
xlabel("Drill Speed (rpm)")
ylabel("Drill Torque (N.m)")
legend("Drill Output", "Given Requirement")
hold off

```



Shaft Torque and Speed Estimation

```

%{
Motor - P1 : Shaft A
N1 - P2    : Shaft B
N2 - Chuck : Shaft C
%}

outputPower = reqOutputSpeed*reqOutputTorque*pi/30 ;

```

```

intermediatePower = outputPower/Efficiency^0.5;
inputPower = intermediatePower/Efficiency^0.5;
efficiency = outputPower/inputPower;

%Shaft C
SpeedC = reqOutputSpeed;
TorqueC = reqOutputTorque;
%Shaft B
SpeedB = SpeedC*m2actual;
TorqueB = intermediatePower/(SpeedB*pi/30);
%Shaft A
SpeedA = SpeedB*m1actual;
TorqueA = inputPower/(SpeedA*pi/30);

TorqueSpeedEstimation = table(TorqueA, SpeedA, TorqueB, SpeedB,
    TorqueC, SpeedC)
PowerTable = table(inputPower, intermediatePower, outputPower,
    efficiency)

```

TorqueSpeedEstimation =

1×6 table

<i>TorqueA</i>	<i>SpeedA</i>	<i>TorqueB</i>	<i>SpeedB</i>	<i>TorqueC</i>	<i>SpeedC</i>
3.0556	11500	14.494	2300	55	575

PowerTable =

1×4 table

<i>inputPower</i>	<i>intermediatePower</i>	<i>outputPower</i>	<i>efficiency</i>
3679.7	3490.9	3311.8	0.9

Operating Gear Forces, Torques and Diameters

```

Forcelt = [0];
Forcelr = [0];
Force2t = [0];
Force2r = [0];

P1diameter = [0];
N1diameter = [0];
P2diameter = [0];
N2diameter = [0];

for x = DiametralPitch

```

```

x = x/0.0254; %Teeth/inch -> teeth/meter
P1dia = P1/x; %Teeth / (Teeth/meter) = meter
N1dia = N1/x;
P2dia = P2/x;
N2dia = N2/x;

P1diameter = [P1diameter, P1dia];
N1diameter = [N1diameter, N1dia];
P2diameter = [P2diameter, P2dia];
N2diameter = [N2diameter, N2dia];

%Transmitted Load: Pinion 1 -> Gear 1
Wt1 = (60000*inputPower*10^-3)/(pi*P1dia*SpeedA);
%Radial Load: Pinion 1 -> Gear 1
Wr1 = tan(PressureAngle)*Wt1;

%Transmitted Load: Pinion 2 -> Gear 2
Wt2 = (60000*outputPower*10^-3)/(pi*N2dia*SpeedC);
%Radial Load: Pinion 2 -> Gear 2
Wr2 = tan(PressureAngle)*Wt2;

Forcelt = [Forcelt, Wt1];
Forcelr = [Forcelr, Wr1];
Force2t = [Force2t, Wt2];
Force2r = [Force2r, Wr2];
end

Forcelt = transpose(Forcelt(2:end));
Force2t = transpose(Force2t(2:end));
Forcelr = transpose(Forcelr(2:end));
Force2r = transpose(Force2r(2:end));

P1diameter = transpose(P1diameter(2:end)).*100; %meters ->
centimeters;
N1diameter = transpose(N1diameter(2:end)).*100; %meters ->
centimeters;
P2diameter = transpose(P2diameter(2:end)).*100; %meters ->
centimeters;
N2diameter = transpose(N2diameter(2:end)).*100; %meters ->
centimeters;

ForceDiameterTable = table(transpose(DiametralPitch), Forcelt,
Forcelr, P1diameter, N1diameter, Force2t, Force2r, P2diameter,
N2diameter)

%{

Not confident that this is correct

%Wt, Tangential Forces
Forcelt = [0];
Force2t = [0];
%Wr, Radial Forces
Forcelr = [0];

```

```

Force2r = [0];
%Gear Diameters
P1diameter = [0];
N1diameter = [0];
P2diameter = [0];
N2diameter = [0];
%Shaft Torques
AT = [0];
BT = [0];
CT = [0];

for x = DiametralPitch
DiametralPitch1 = x/0.0254;
DiametralPitch2 = x/0.0254; %Teeth/inch -> teeth/meter

P1dia = P1/DiametralPitch1; %Teeth / (Teeth/meter) = meter
N1dia= N1/DiametralPitch1;
P2dia= P2/DiametralPitch2;
N2dia= N2/DiametralPitch2;

%Logging diameters for the table below
P1diameter = [P1diameter, P1dia];
N1diameter = [N1diameter, N1dia];
P2diameter = [P2diameter, P2dia];
N2diameter = [N2diameter, N2dia];

%Tangential Forces
Ctorque = CopTorque;
Wt2 = Ctorque/(cos(PressureAngle2)*N2dia/2);
Btorque = (Wt2*P2dia/2)/cos(PressureAngle2);
Wt1 = Btorque/(cos(PressureAngle1)*N1dia/2);
Atorque = (Wt1*P1dia/2)/cos(PressureAngle1);
Force2t = [Force2t, Wt2];
Forcelt = [Forcelt, Wt1];
%Radial Forces
Forcelr = [Forcelr, tan(PressureAngle1)*Wt1];
Force2r = [Force2r, tan(PressureAngle2)*Wt2];
%Shaft Torques
AT = [AT, Atorque];
BT = [BT, Btorque];
CT = [CT, Ctorque];

end

Forcelt = Forcelt(2:end);
Force2t = Force2t(2:end);
Forcelr = Forcelr(2:end);
Force2r = Force2r(2:end);

P1diameter = P1diameter(2:end);
N1diameter = N1diameter(2:end);
P2diameter = P2diameter(2:end);
N2diameter = N2diameter(2:end);

```

```

AT = AT(2:end);
BT = BT(2:end);
CT = CT(2:end);

DiametralPitch = transpose(DiametralPitch);
Forcelt = transpose(Forcelt);
Force2t = transpose(Force2t);
Forcelr = transpose(Forcelr);
Force2r = transpose(Force2r);
P1diameter = transpose(P1diameter).*100; %meters -> centimeters
N1diameter = transpose(N1diameter).*100;
P2diameter = transpose(P2diameter).*100;
N2diameter = transpose(N2diameter).*100;
AT = transpose(AT);
BT = transpose(BT);
CT = transpose(CT);

ForceTorqueTable = table(DiametralPitch, AT, Forcelt, Forcelr, BT,
    Force2t, Force2r, CT)

DiameterTable = table(DiametralPitch, P1diameter, N1diameter,
    P2diameter, N2diameter)

%}

ForceDiameterTable =

8x9 table

    Var1    Forcelt    Forcelr    P1diameter    N1diameter    Force2t
Force2r    P2diameter    N2diameter
-----
      8      120.3      43.785          5.08          25.4      541.34
197.03          5.08          20.32
     10      150.37      54.731          4.064          20.32      676.67
246.29          4.064          16.256
     12      180.45      65.677          3.3867          16.933      812.01
295.55          3.3867          13.547
     16      240.59      87.569          2.54          12.7      1082.7
394.06          2.54          10.16
     20      300.74      109.46          2.032          10.16      1353.3
492.58          2.032          8.128
     24      360.89      131.35          1.6933          8.4667      1624
591.09          1.6933          6.7733
     32      481.19      175.14          1.27          6.35      2165.4
788.12          1.27          5.08
     48      721.78      262.71          0.84667          4.2333      3248
1182.2          0.84667          3.3867

```

Stresses

(Calculated in USCS then converted to metric)

```
ContactPinion1 = [0];
BendingPinion1 = [0];
ContactGear1 = [0];
BendingGear1 = [0];
ContactPinion2 = [0];
BendingPinion2 = [0];
ContactGear2 = [0];
BendingGear2 = [0];

for x = [1:length(DiametralPitch)]
    %Pinion 1
    [CP1, BP1] = stresses(DiametralPitch(x), P1, SpeedA, FaceWidth(x),
        Forcelt(x), mlactual, PressureAngle);
    %Gear 1
    [CN1, BN1] = stresses(DiametralPitch(x), N1, SpeedB, FaceWidth(x),
        Forcelt(x), mlactual, PressureAngle);
    %Pinion 2
    [CP2, BP2] = stresses(DiametralPitch(x), P2, SpeedB, FaceWidth(x),
        Force2t(x), m2actual, PressureAngle);
    %Gear 2
    [CN2, BN2] = stresses(DiametralPitch(x), N2, SpeedC, FaceWidth(x),
        Force2t(x), m2actual, PressureAngle);

    ContactPinion1 = [ContactPinion1, CP1];
    BendingPinion1 = [BendingPinion1, BP1];
    ContactGear1 = [ContactGear1, CN1];
    BendingGear1 = [BendingGear1, BN1];
    ContactPinion2 = [ContactPinion2, CP2];
    BendingPinion2 = [BendingPinion2, BP2];
    ContactGear2 = [ContactGear2, CN2];
    BendingGear2 = [BendingGear2, BN2];

end

ContactPinion1 = transpose(ContactPinion1(2:end));
BendingPinion1 = transpose(BendingPinion1(2:end));
ContactGear1 = transpose(ContactGear1(2:end));
BendingGear1 = transpose(BendingGear1(2:end));
ContactPinion2 = transpose(ContactPinion2(2:end));
BendingPinion2 = transpose(BendingPinion2(2:end));
ContactGear2 = transpose(ContactGear2(2:end));
BendingGear2 = transpose(BendingGear2(2:end));

StressTable = table(transpose(DiametralPitch), ContactPinion1,
    BendingPinion1, ContactGear1, BendingGear1, ContactPinion2,
    BendingPinion2, ContactGear2, BendingGear2)
```

```

%{
Pd = Diametral Pitch
N = Number of Teeth on pinion
n = Angular Velocity (rpm)
F = Face Width
Wt = Tangential load
m = gear Ratio
phi = pressure angle
ht = tooth height
tr = rim thickness
%}
function [CS, BS] = stresses(Pd, N, n, F, Wt, m, phi)
    dp = N/Pd;           %Diameter inches
    V = pi*dp*n/12;       %Linear Velocity

    %Ko - Overload Factor
    Ko = 1.2; %Moderate Shock

    %Kv - Dynamic Factor
    Qv = 9; %Low end of precision quality gears, Catalog: pg. 48, AGMA
class: 9
    B = 0.25*(12 - Qv)^(2/3);
    A = 50 + 56*(1 - B);
    Kv = ((A + V^0.5)/A)^B;

    %Ks - Size Factor
    x = 1.5708/Pd; %Tooth Width, equation from boston
    Y = 2*x*Pd/3;
    Ks = 1.192*((F*Y^0.5)/Pd)^0.0535;

    %Km - Load Distribution Factor
    Cmc = 0.8; %Crowned
    Cpf = F/(10*dp) - 0.025;
    Cpm = 1;
    A = 0.0675; %Precision, enclosed, Table 14-9
    B = 0.0128;
    C = -0.926E-4;
    Cma = A + B*F + C*F^2;
    Ce = 1;
    Km = 1 + Cmc*(Cpf*Cpm + Cma*Ce);

    %Kb - Rim Thickness Factor
    Kb = 1; %Assumption given by professor
    %{
    if Pd < 20 %Whole Depth Equation from Boston
        ht = 2.157/Pd;
    else
        ht = 2.2/Pd + 0.002;
    end

    tr = %Cannot find equation or values specifying rim thickness

    mb = tr/ht;

```

```

if mb < 1.2
    Kb = 1.6*log(2.242/mb);
else
    Kb = 1;
end
%}

%J - Bending Strength Geometry Factor
J = 0.275; %Figure 14-6

%Kt - Temperature Factor
Kt = 1;

%Kr - Reliability Factor
Kr = 1; %0.99 - Eq. 14-38

%Cp - Elastic Coefficient
Cp = 2300; %Table 14-8 steel-steel

%I - Pitting Resistance Geometry Factor
mn = 1; %Load sharing ratio: 1 for spur gears
I = ((cos(phi)*sin(phi))/(2*mn))*(m/(m+1));

%Cf - surface condition factor
Cf = 1;

%Zn - Stress-Cycle Factor
Zn = 1;

%Contact Stress
CS = (Cp*sqrt(Wt*Ko*Kv*Ks*(Km/(dp*F)))*(Cf/I)))*0.00689476; %Contact stress in psi -> MPa

%Bending Stress
BS = (Wt*Ko*Kv*Ks*(Pd/F)*((Km*Kb)/J))*0.00689476; %Contact stress
in psi -> MPa
end

```

StressTable =

8×9 table

Var1	ContactPinion1	BendingPinion1	ContactGear1
BendingGear1	ContactPinion2	BendingPinion2	ContactGear2
BendingGear2			
8	429.8	39.46	188.67
38.021	855.07	149.94	420.16
144.81			

10	584.51	72.982	256.58
70.314	1169.3	280.37	574.53
270.76			
12	788.34	132.76	346.67
128.36	1583.9	514.5	779.57
498.52			
16	1239	327.95	545.84
318.22	2506.2	1288	1235.5
1252.2			
20	1740.6	647.17	767.47
629.12	3538.1	2567.2	1745.8
2500			
24	2485.5	1319.7	1099
1290	5072.1	5275.7	2509.1
5164.4			
32	3713.3	2945.5	1641.8
2879	7622.1	11914	3770.5
11662			
48	6572.1	9226.7	2905.8
9018.9	13593	37891	6724.3
37090			

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