

Duffing Oscillator

Ph235 Spring 2019

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Problem statement

- Initial: Model chaotic motion of a magnetoelastic steel beam due to magnetic forces and elasticity
- **Modified** → examine and identify points of bifurcation through visuals for non-obvious duffing oscillator system

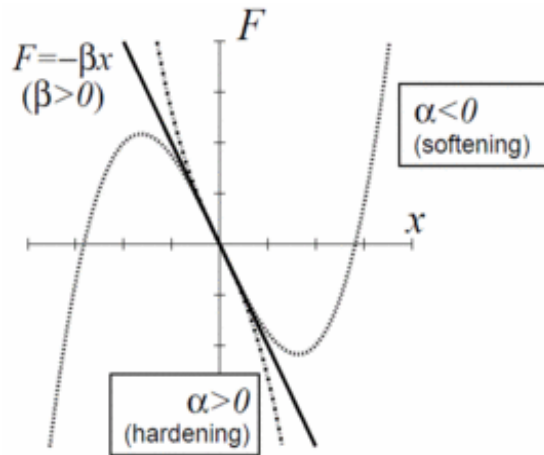


Figure 1. For $\beta > 0$: a forced oscillator model with a spring whose restoring force is written as $F = -\beta x - \alpha x^3$

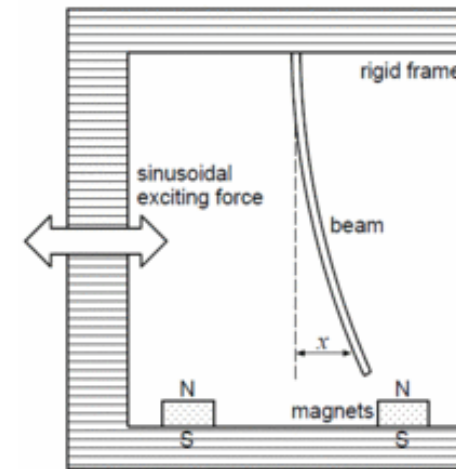


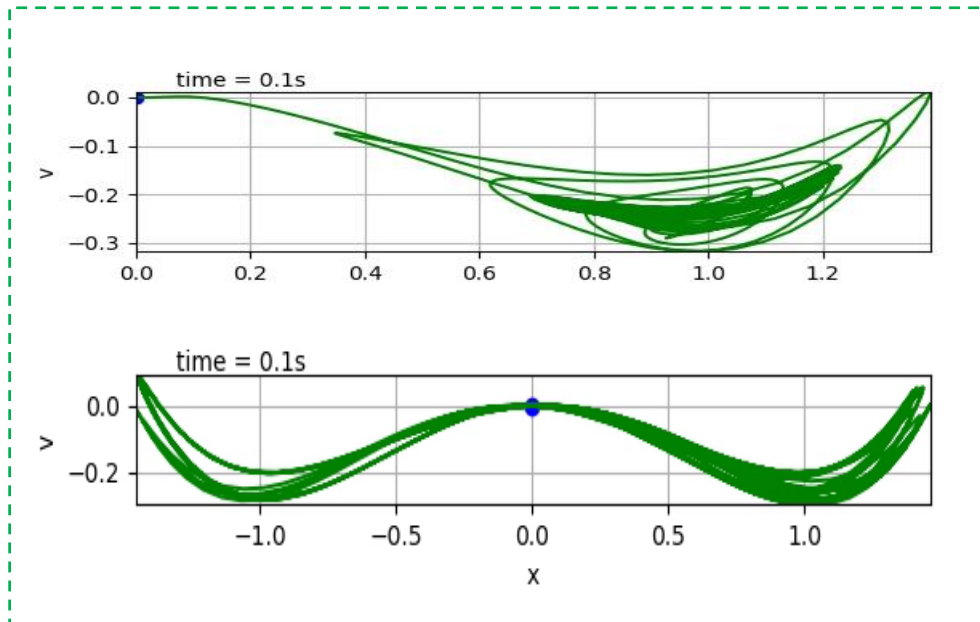
Figure 2. For $\beta < 0$ a model of a periodically forced steel beam which is deflected toward the two magnets.
(Moon and Holmes, 1979; Guckenheimer and Holmes, 1983; Ott, 2002)

Duffing's Equation

- Nonlinear 2nd order ODE used to model damped driven systems

$$\ddot{x} + \delta\dot{x} + \alpha x + \beta x^3 = \Upsilon \cos(\omega t)$$

where $x = \text{displacement at time } t$,
 $\delta = \text{damping coefficient}$,
 $\alpha = \text{linear stiffness coefficient}$
 $\beta = \text{nonlinearity in the restoring force}$
 $\Upsilon = \text{amplitude of } \textit{excitation}$
 $\omega = \text{angular frequency of excitation}$



Stability Check

- Eigenvalues of Jacobian Matrix of undamped, unforced system

$$\ddot{x} + \delta \dot{x} + \alpha x + \beta x^3 = \gamma \cos(\omega t) \dots (1)$$

$$x(\alpha + \beta x^2) = 0 \dots (2)$$

$$\rightarrow x = 0; \quad x = \pm \sqrt{\frac{-\alpha}{\beta}}$$

For $\delta=0$, one 2D \rightarrow two 1D equations

$$\frac{d}{dt} \begin{pmatrix} x \\ \dot{x} \end{pmatrix} = \begin{pmatrix} \dot{x} \\ -\delta \dot{x} - \alpha x - \beta x^3 \end{pmatrix}$$

$$J(x) = \begin{pmatrix} 0 & 1 \\ -\alpha - 3\beta x^2 & -\delta \end{pmatrix}$$

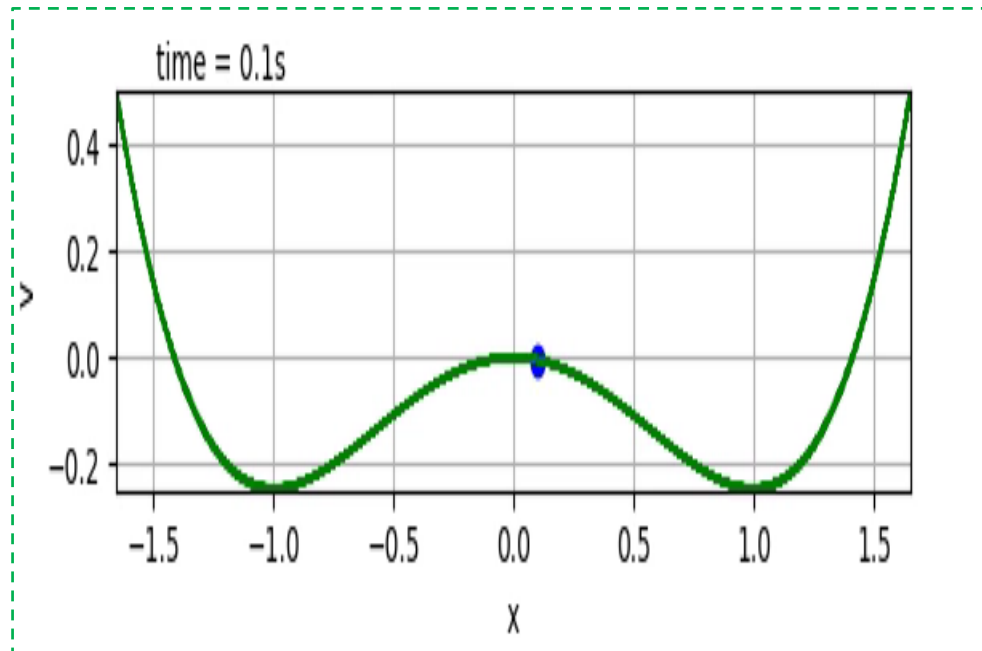
$$\lambda_{eig} = \frac{-\delta \pm \sqrt{\delta^2 - 4\alpha}}{2}$$

$$x = 0$$

$$\lambda_{eig} = \frac{-\delta \pm \sqrt{\delta^2 + 8\alpha}}{2}$$

$$x = \pm \sqrt{\frac{-\alpha}{\beta}}$$

Stability Check



$$\lambda_{eig} = \frac{-\delta \pm \sqrt{\delta^2 - 4\alpha}}{2}$$

$$\lambda_{eig} = \frac{-\delta \pm \sqrt{\delta^2 + 8\alpha}}{2}$$

$$\delta = 0, \quad \alpha = -1:$$

$$\text{at } x = 0, \quad \lambda_{eig} = 1$$

$$\text{at } x = \pm 1, \quad \lambda_{eig} \approx 0$$

Numerical Approximation

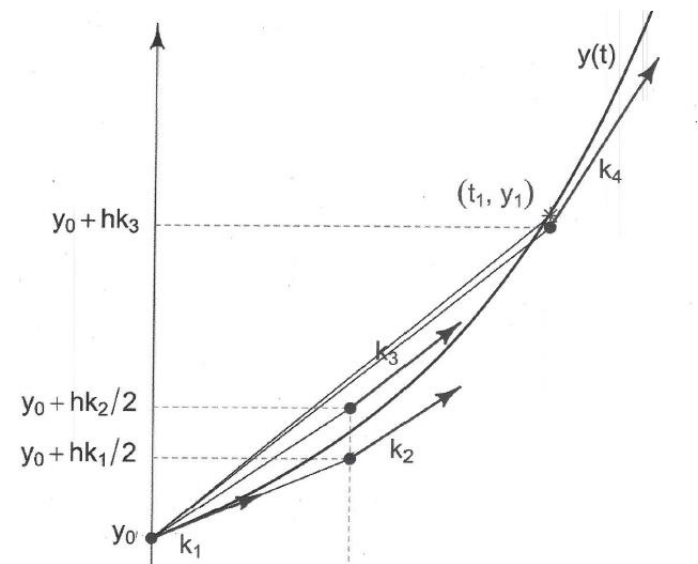


Figure 3. RK4 Method

- Fourier Series, Frobenius method, Homotopy analysis method (HAM)

- **4th order Runge-Kutta Method**

- application:

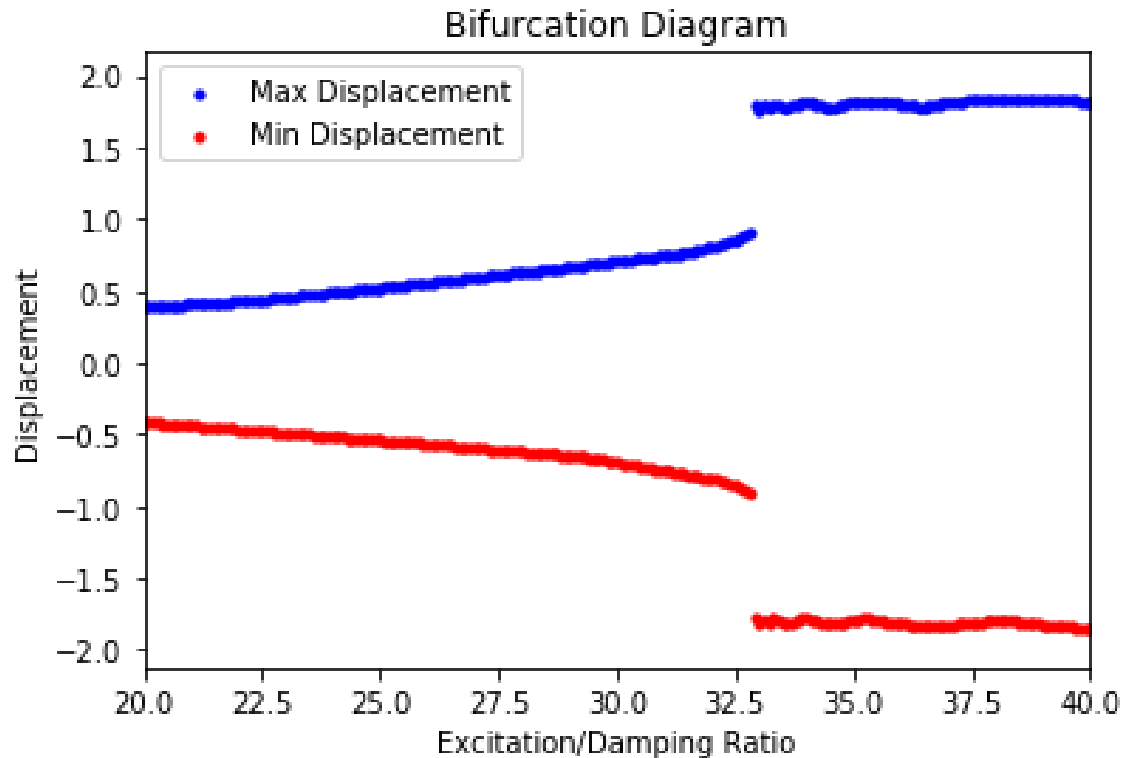
$$m \frac{d^2x}{dt^2} + \delta \frac{dx}{dt} + \alpha x + \beta x^3 = F(t)$$

$$\begin{aligned} \dot{x} &= v \\ \dot{v} &= -\delta v - \alpha x - \beta x^3 \\ \dot{\theta} &= w \end{aligned}$$

TABLE 1 FOURTH-ORDER RUNGE-KUTTA IMPLEMENTATION VERIFICATION^[1]

Step size (h)	Approximation of x at t = 0.1, x _{approx}	Final Global Error(F.G.E) at t = 0.1 (x _{approx} - x _{true})	Error Ratio
0.1	-0.0018921715207243900	-0.0000000076565361900	-
0.05	-0.0018921642988212700	-0.0000000004346330700	17.6
0.025	-0.0018921638899839600	-0.0000000000257957601	16.8
0.0125	-0.0018921638657577500	-0.0000000000015695500	16.4
0.0001	-0.0018921638641882000 (=x _{true})	(assume true value)	-

Bifurcation



- Def: Point where the change in the parameter values results in a change in the stability at the equilibrium point(s).

- Divergence at $\frac{\gamma}{\delta} \sim 33.0$

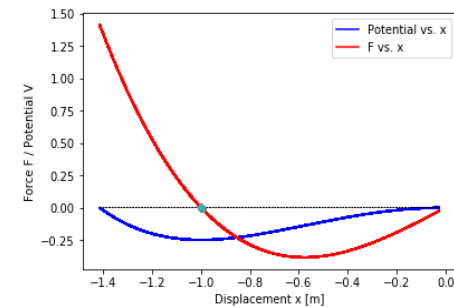
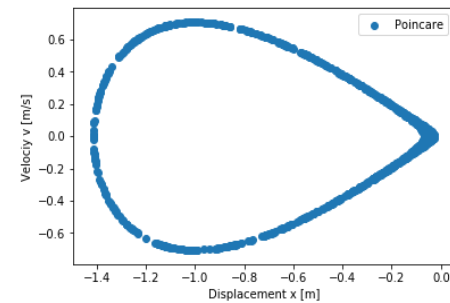
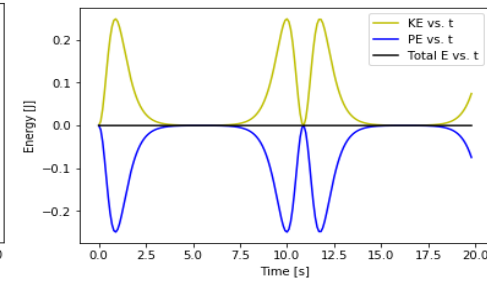
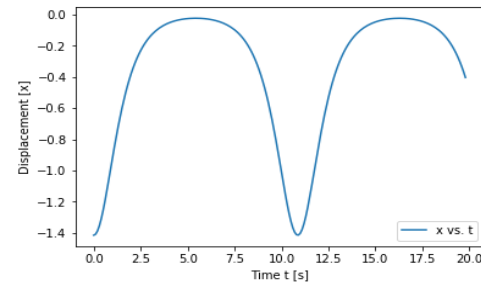
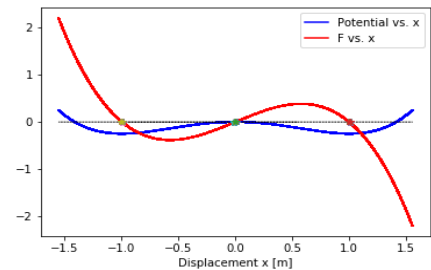
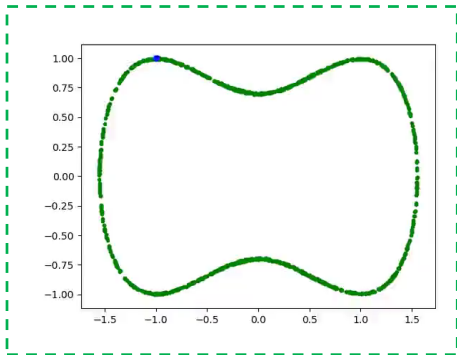
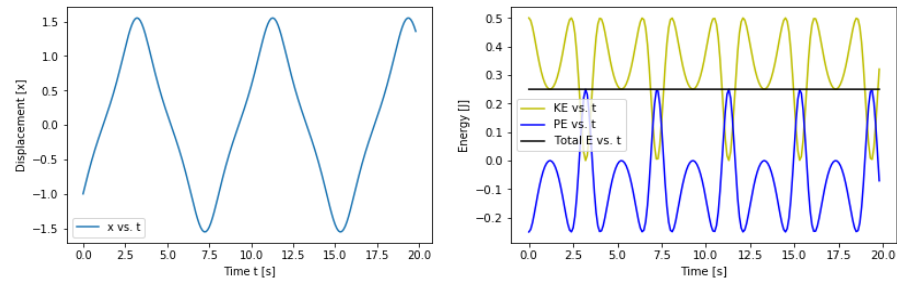
Initial conditions : $x=0, v=0$

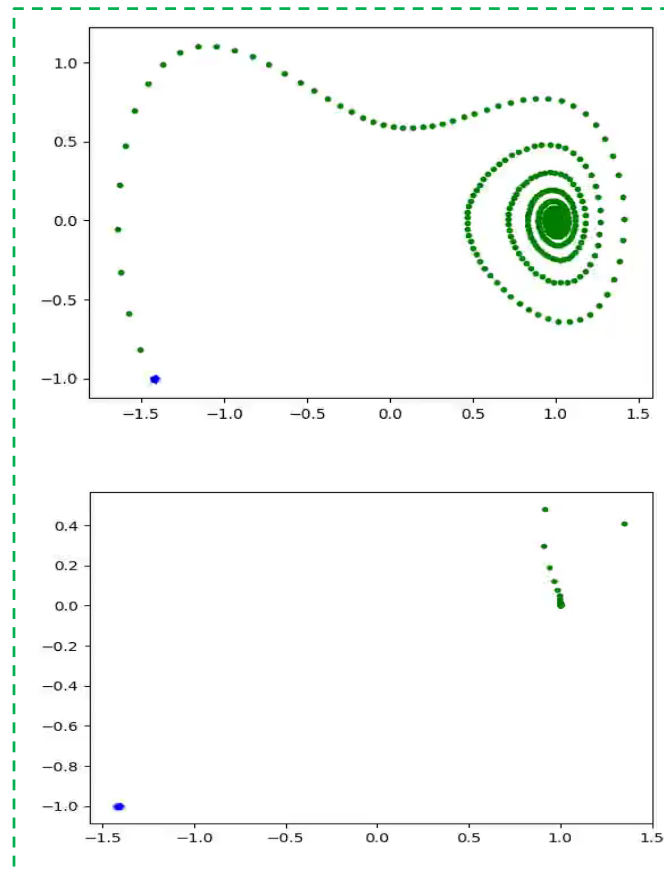
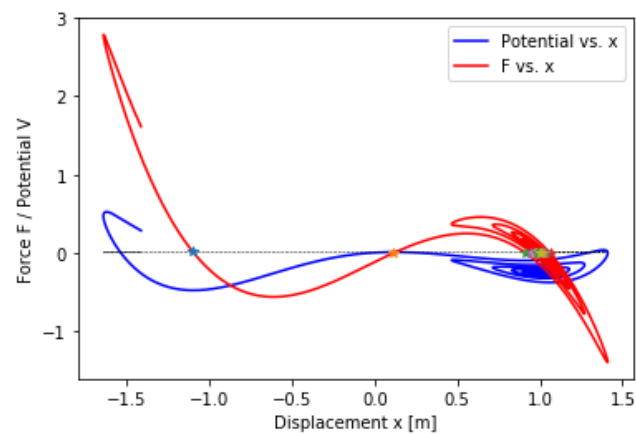
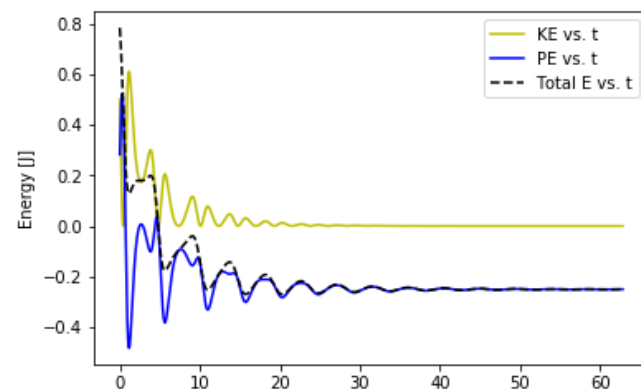
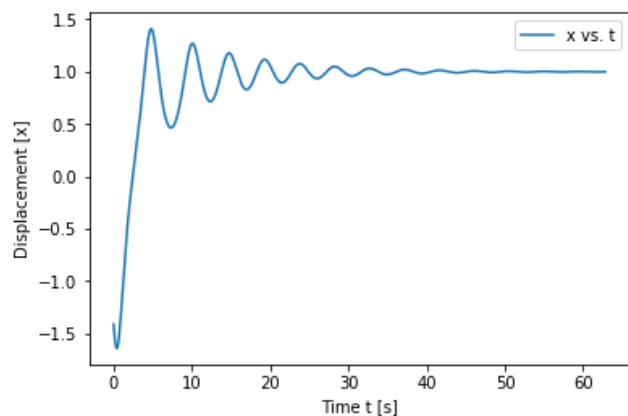
$\delta = 0.01, \alpha = 1, \beta = 1, w = 1.414,$
 $h = 0.01$

Free motion

$$\delta = 0, \alpha = -1, \beta = 1, \gamma = 0,$$

$$x = -1, v = 1$$





Damped Motion

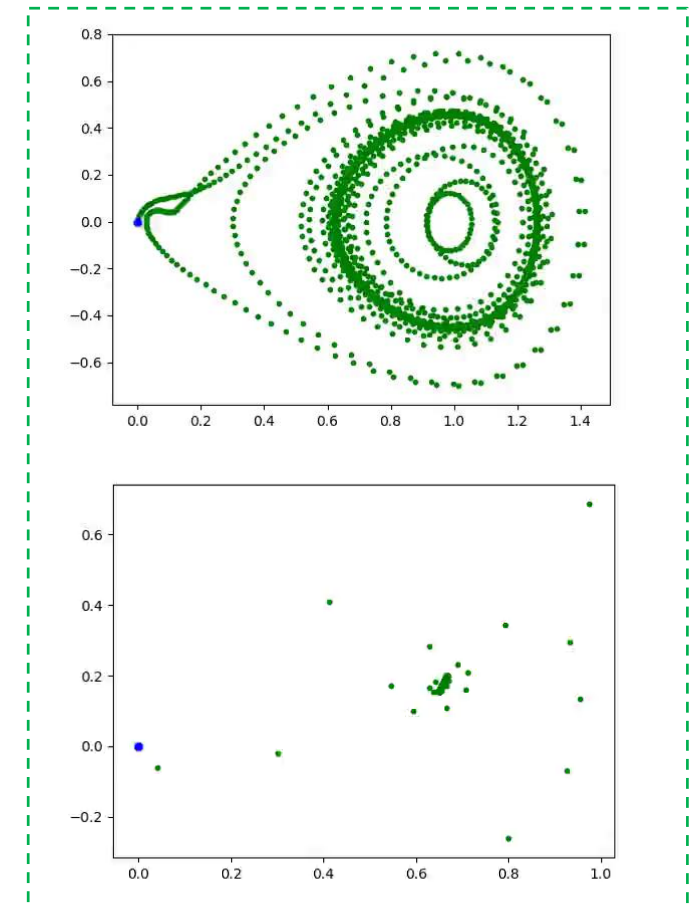
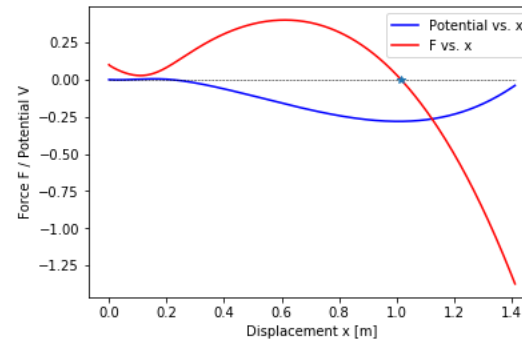
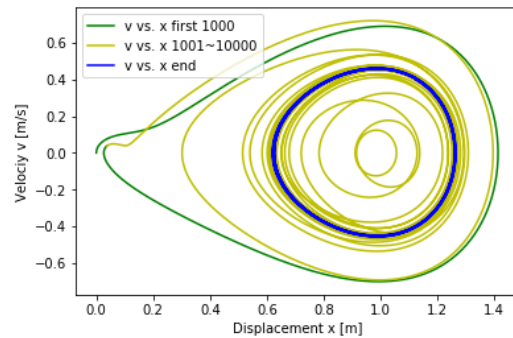
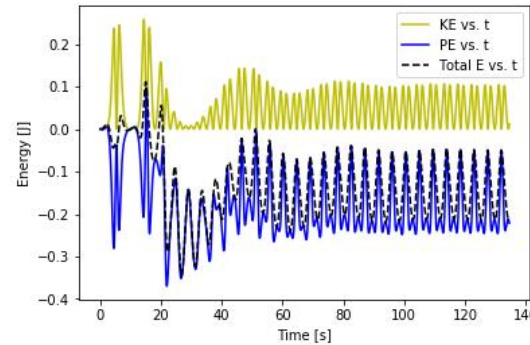
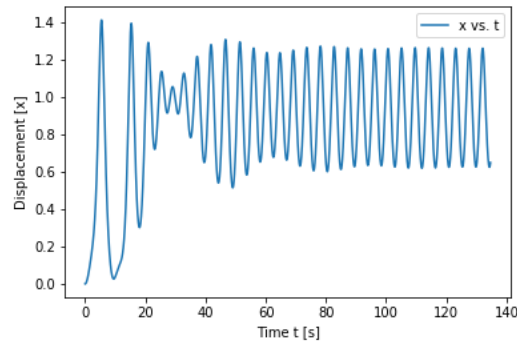
$$\delta = 0.2, \alpha = -1, \beta = 1, \gamma = 0,$$

$$x = -1.414, v = -1$$

Damped Driven Motion

$$\delta = 0.1, \alpha = -1, \beta = 1, \gamma = \mathbf{0.1}, \omega = 1.4$$

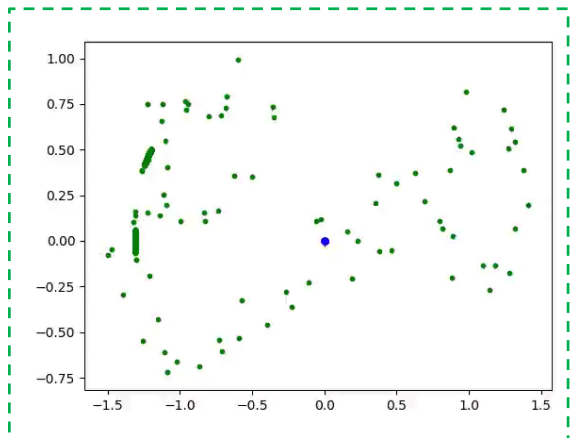
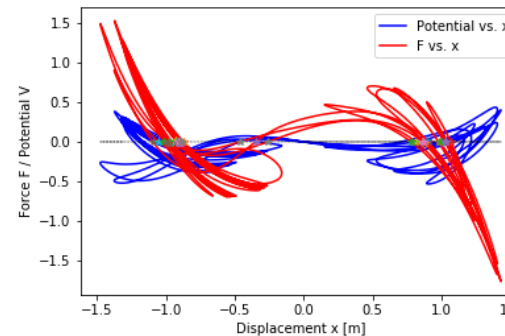
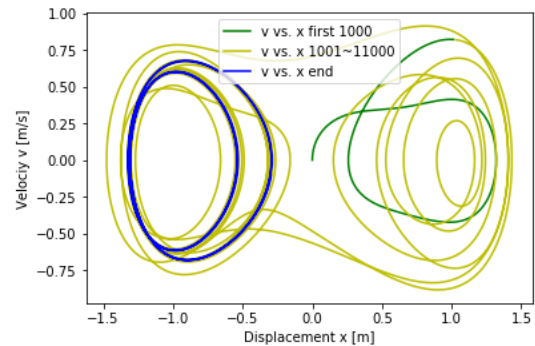
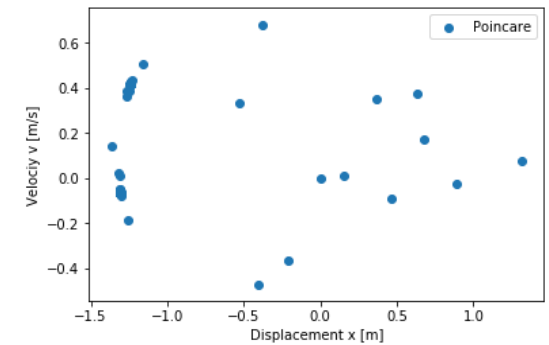
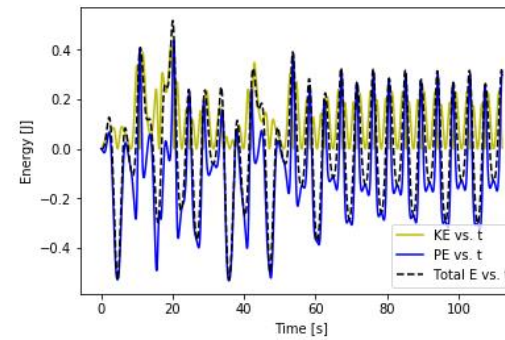
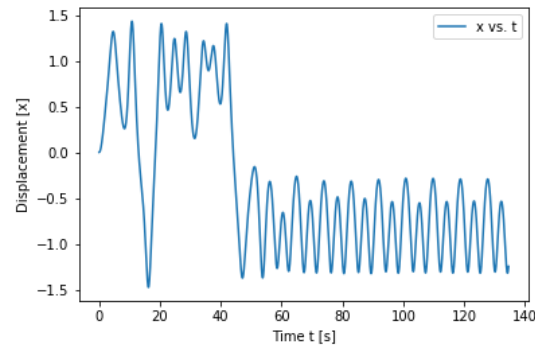
$$x = 0, v = 0$$



Damped Driven Motion2

$$\delta = 0.1, \alpha = -1, \beta = 1, \gamma = \mathbf{0.32}, \omega = 1.4$$

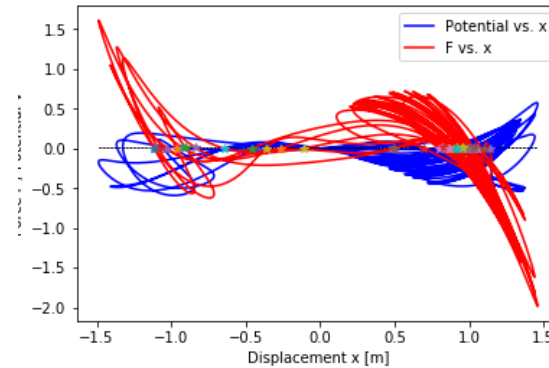
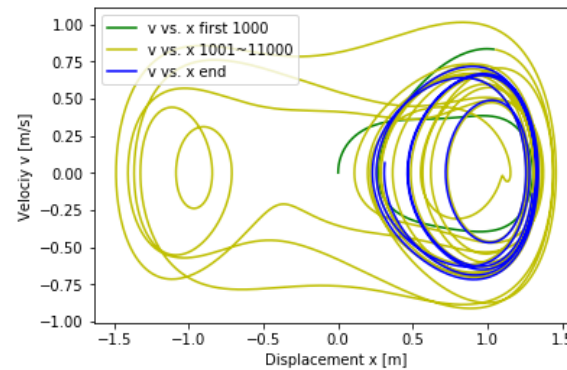
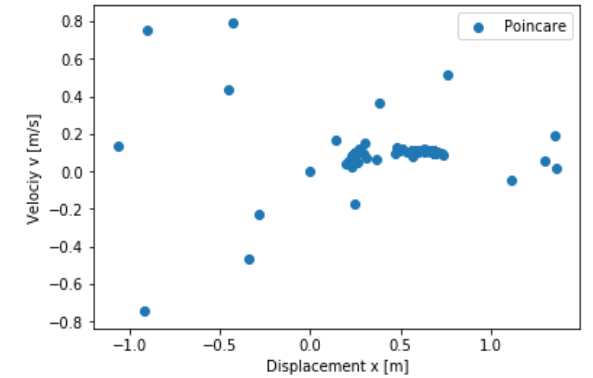
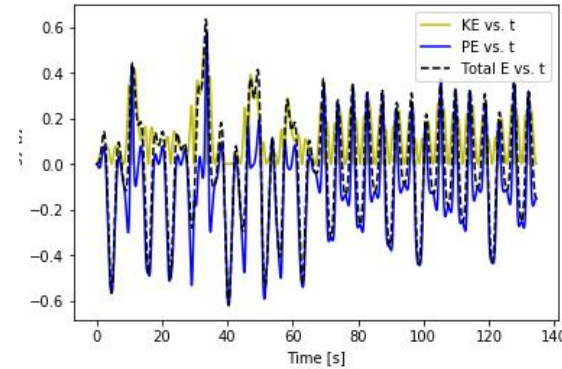
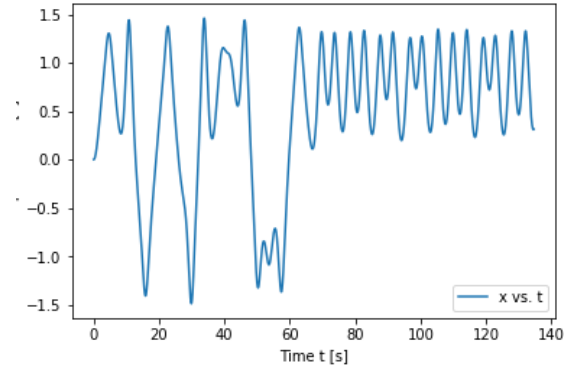
$$x = 0, v = 0$$



Damped Driven Motion3

$$\delta = 0.1, \alpha = -1, \beta = 1, \gamma = \mathbf{0.34}, \omega = 1.4$$

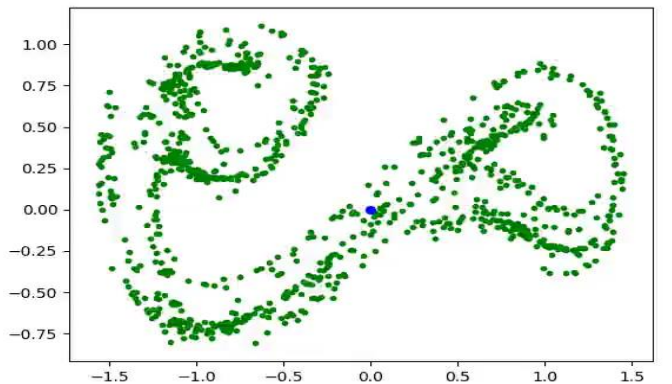
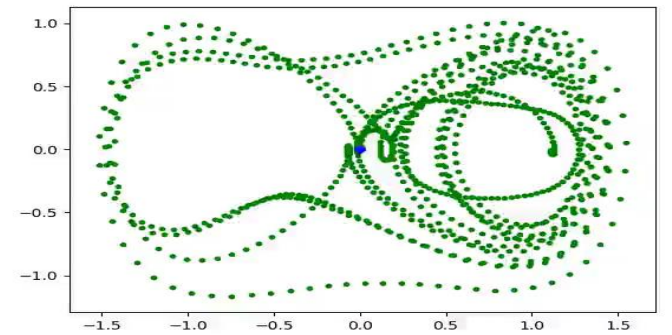
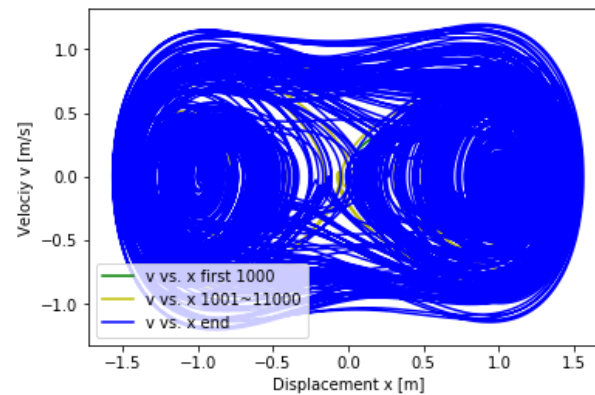
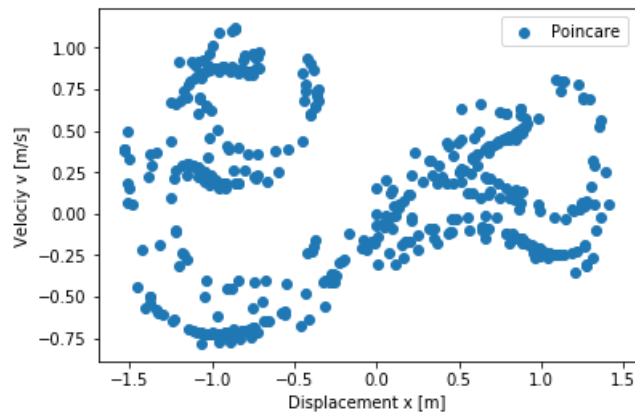
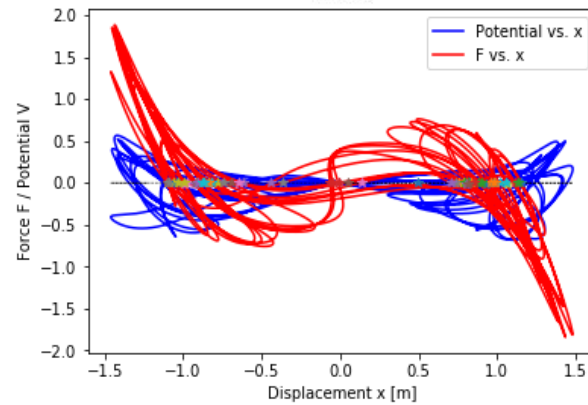
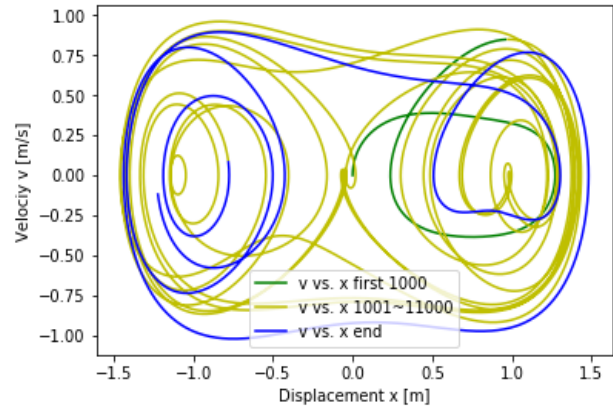
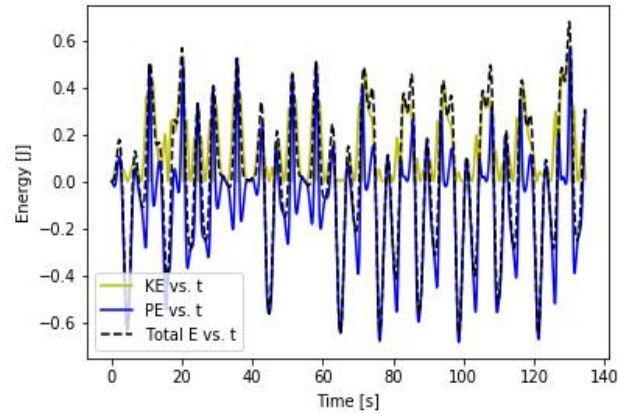
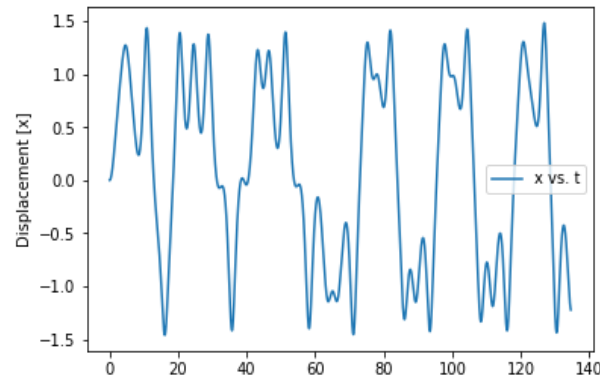
$$x = 0, v = 0$$



Damped Driven Motion₄: Chaos!

$$\delta = 0.1, \alpha = -1, \beta = 1,$$

$$\gamma = 0.38, \omega = 1.4, x = 0, v = 0$$



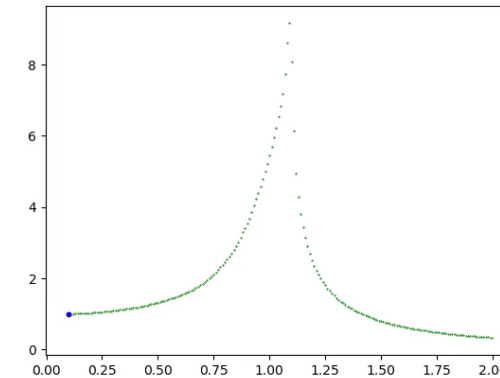
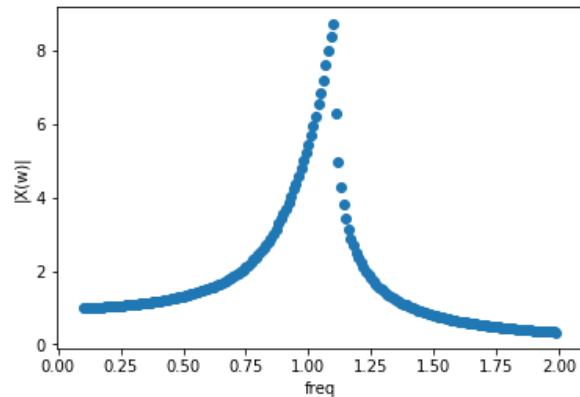
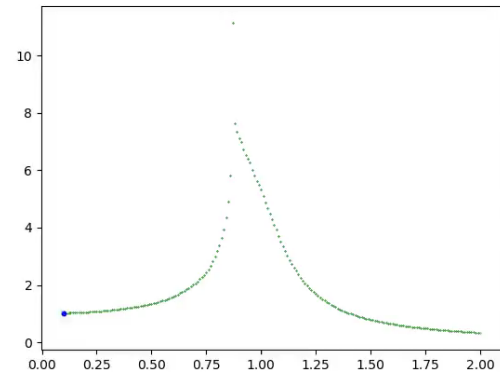
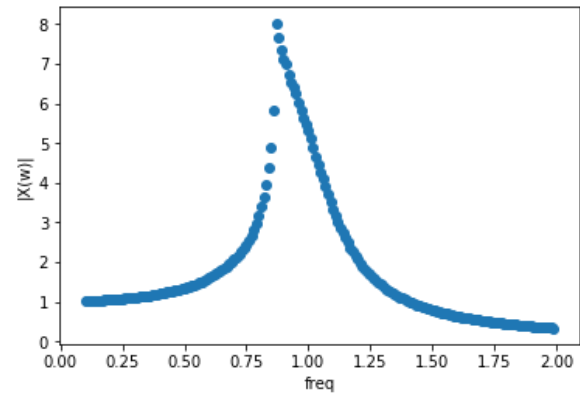
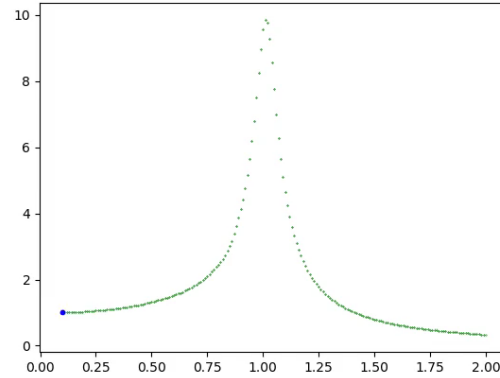
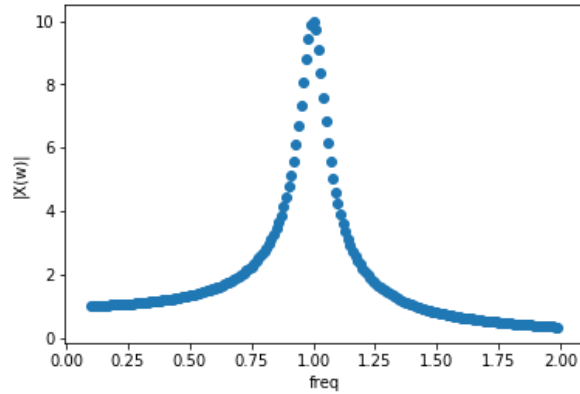
Frequency Response

* harmonic balance method

(i) Linear System (point of bifurcation)

(ii) Softening System

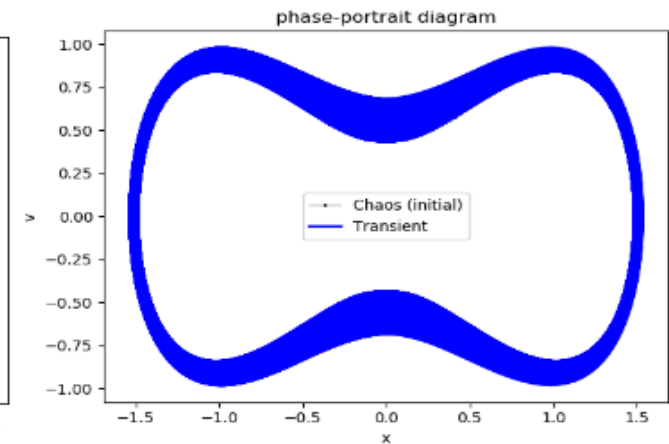
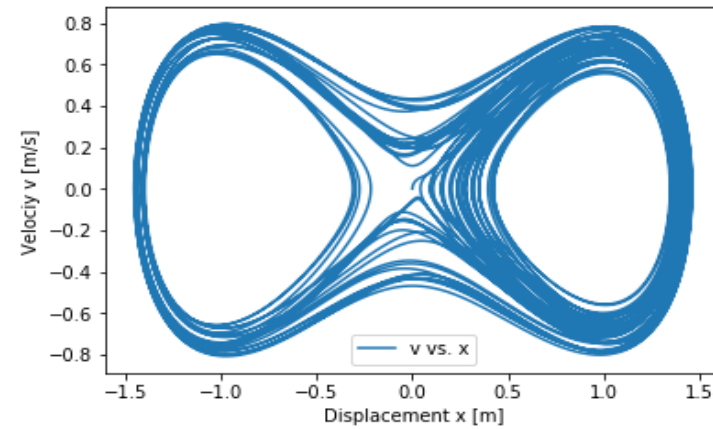
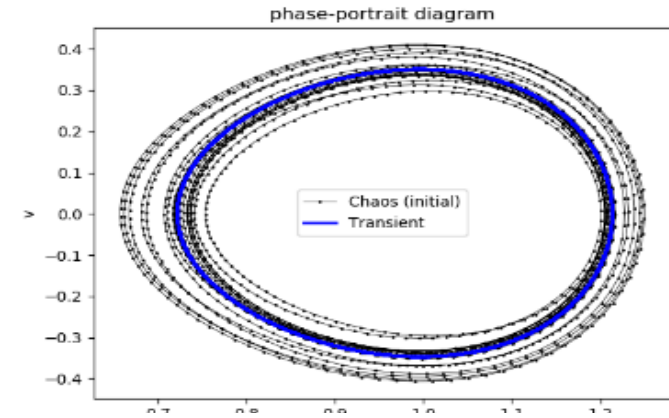
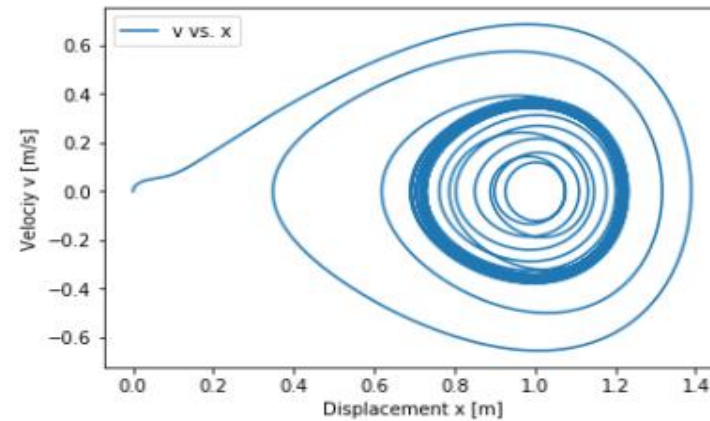
(iii) Hardening system



Built-in 'Odeint'

(i) Single-Well

(ii) Double-Well



Visual (1D)

