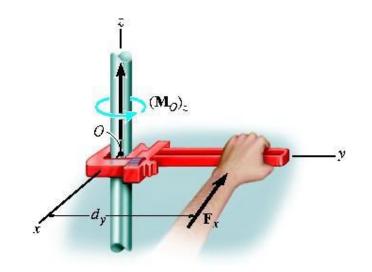
Chapter 4: Force System Resultants

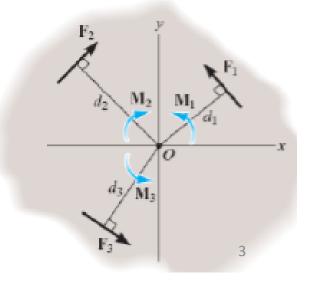
Plan

- Moment of a force-Scalar formulation
- Cross product
- Moment of a force-Vector formulation
- Principle of moments
- Moment of a force about a specified axis
- Moment of a couple
- Simplification of a force and couple system
- Reduction of a simple distributed loading

Moment of a force-Scalar formulation

- Moment of a force: tendency of a force to rotate a rigid body about any defined axis.
- Torque or twist moment: tendency of rotation caused by F_x or simple moment $(M_O)_z$
- Magnitude M_0 =F.d (N.m) where d: perpendicular distance from O to its line of action
- Direction using "right hand rule": positive
 CCW and negative CW
- Resultant moment M_{Ro}=ΣFd





Cross Product

Cross product of vectors A and B yields vector C, written as:

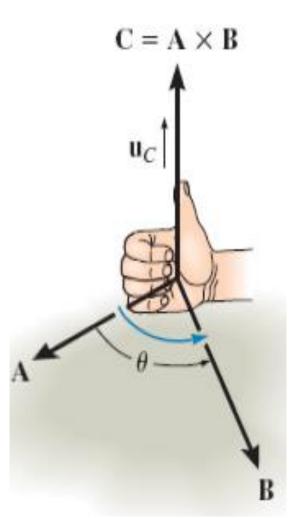
$$C=A \times B=(AB \sin \theta)u_C$$

- Law of operations:
- 1. Commutative law is not valid

$$A \times B \neq B \times A$$
; $A \times B = -B \times A$

- 2. Multiplication by a scalar $a(\mathbf{A} \times \mathbf{B}) = (a\mathbf{A}) \times \mathbf{B} = \mathbf{A} \times (a\mathbf{B}) = (\mathbf{A} \times \mathbf{B})a$
- 3. Distributive law $A \times (B+D)=(A \times B)+(A \times D)$

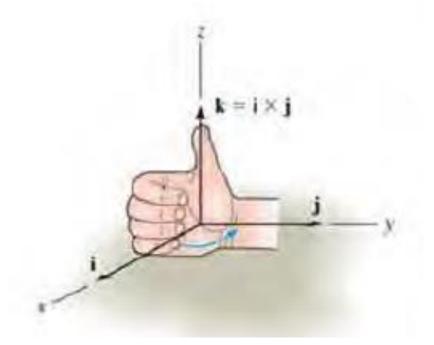
Proper order of the cross product must be maintained

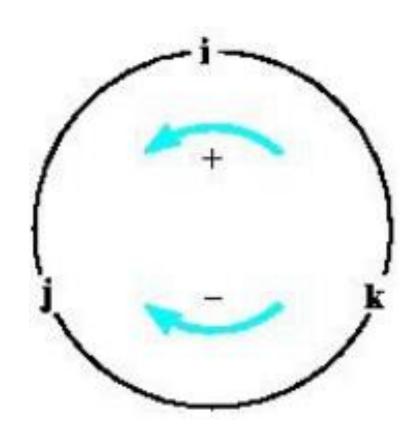


Cross Product

Cartesian Vector Formulation

■ Determinant form $\mathbf{A} \times \mathbf{B} = \begin{bmatrix} \vec{l} & \vec{J} & \vec{k} \\ A_x & A_y & A_z \\ B_x & B_y & B_z \end{bmatrix}$



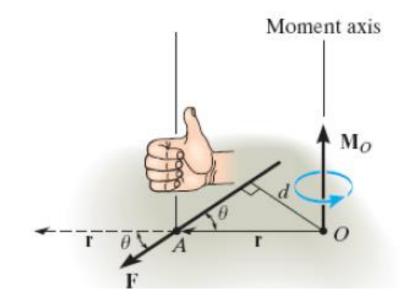


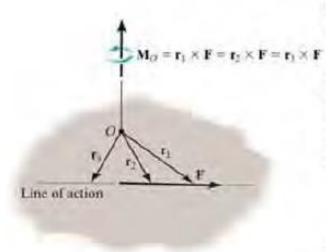
Moment of a force-Vector formulation

- Moment of a force using a cross product
 $M_o = r x F$
- Magnitude of cross product $M_0 = rF \sin\theta$
- With d=r sin $\theta \Rightarrow M_0$ =rF sin θ =F (r sin θ)=Fd



$$\boldsymbol{M_o} = \boldsymbol{r}_1 \times \boldsymbol{F} = \boldsymbol{r}_2 \times \boldsymbol{F} = \boldsymbol{r}_3 \times \boldsymbol{F}$$





Moment of a force-Vector formulation

Cartesian Vector Formulation

 For force F and position vector r expressed in Cartesian form,

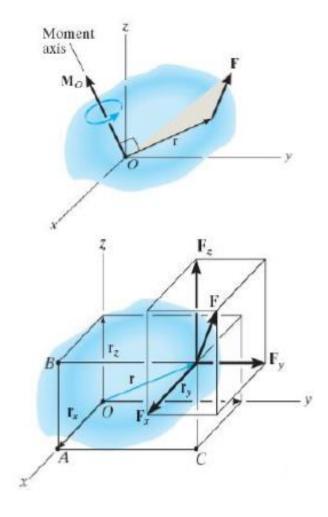
$$\mathbf{M_0} = \mathbf{r} \times \mathbf{F} = \begin{vmatrix} \vec{l} & \vec{J} & \vec{k} \\ r_x & r_y & r_z \\ F_x & F_y & F_z \end{vmatrix}$$



$$\mathbf{M_0} = (r_y F_z - r_z F_y) \mathbf{i} - (r_x F_z - r_z F_x) \mathbf{j} + (r_x F_y - r_y F_x) \mathbf{k}$$

With r_x , r_y , r_z : the x, y, z components of the position vector drawn from point O to any point of the line of action of the force.



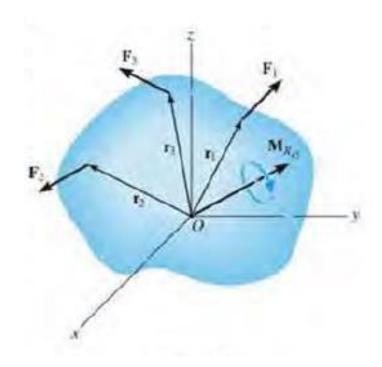


Moment of a force-Vector formulation

Resultant moment of a system of forces

■ The resultant moment of a system of forces about point O can be determined by vector addition of the moment of each force:

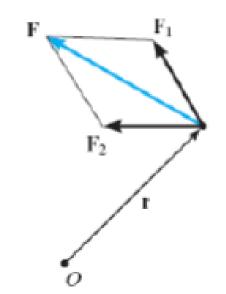
$$M_0 = \Sigma (r \times F)$$

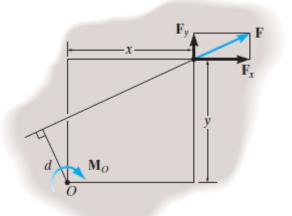


Principle of moments

Varignon's Theorem

- The moment of a force about a point is equal to the sum of the moments of the components of the force about the point.
- Since $F=F_1+F_2$, Mo=r x F=r x (F_1+F_2) =r x F_1 + r x F_2
- For two-dimensional problems, $M_0=F_xy-F_yx$

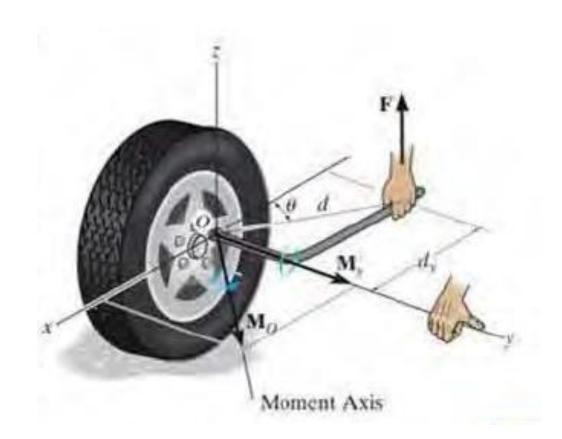




Moment of a force about a specified axis

Scalar Analysis

- The moment of **F** about y axis $M_y = F d_y = F(d \cos \theta)$
- For any axis a, M_a=F d_a



Moment of a force about a specified axis

Vector Analysis

- For magnitude of M_a , $M_a = M_O \cos\theta = M_O \cdot u_a$ with u_a : unit vector
- In determinant form:

$$|\mathbf{M}_{a}| = \mathbf{u}_{a} \cdot (\mathbf{r} \times \mathbf{F}) = \begin{vmatrix} u_{ax} & u_{ay} & u_{az} \\ r_{x} & r_{y} & r_{z} \\ F_{x} & F_{y} & F_{z} \end{vmatrix}$$
Axis of projection

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Moment of a couple

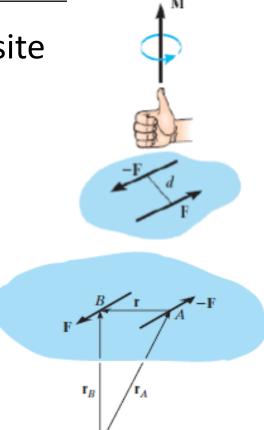
 Couple: two parallel forces of the same magnitude but opposite direction separated by perpendicular distance d



Magnitude of couple moment M=Fd
M acts perpendicular to plan containing the forces

■ Vector formulation For couple moment, $\mathbf{M} = \mathbf{r}_{B} \times \mathbf{F} + \mathbf{r}_{A} \times (-\mathbf{F}) = (\mathbf{r}_{B} - \mathbf{r}_{A}) \times \mathbf{F}$ $\mathbf{M} = \mathbf{r} \times \mathbf{F}$

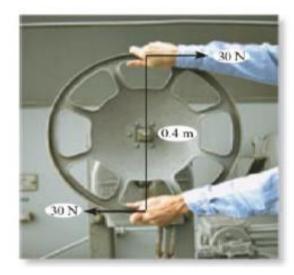
The couple moment is a free vector, it can act at any point since M depends only upon the position vector **r** directed between the forces.

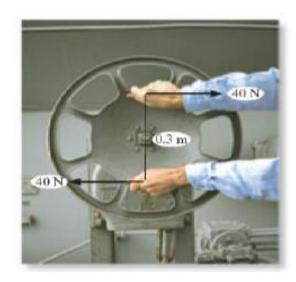


Moment of a couple

Equivalent Couples

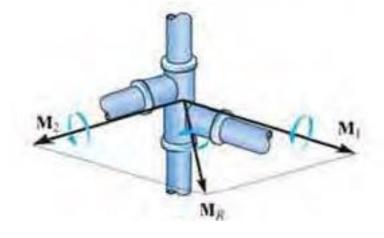
Two couples are equivalent if they produce the same moment Forces of equal couples lie on the same plane or plane parallel to one another





Resultant couple moment:

$$\mathbf{M}_{\mathsf{R}} = \mathbf{\Sigma} \mathbf{r} \mathbf{x} \mathbf{F}$$



Simplification of a force and couple system

 Equivalent resultant force acting at point O and a resultant couple moment is expressed as

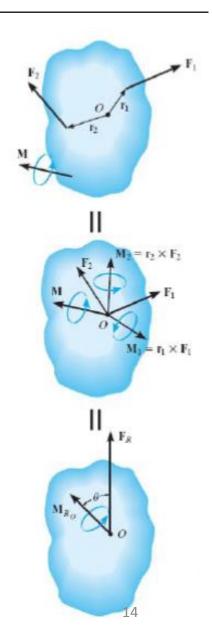
$$F_R = \Sigma F$$

 $(M_R)_O = \Sigma M_O + \Sigma M$

If the force system lies in the x-y plane, the couple moments are perpendicular to this plane :

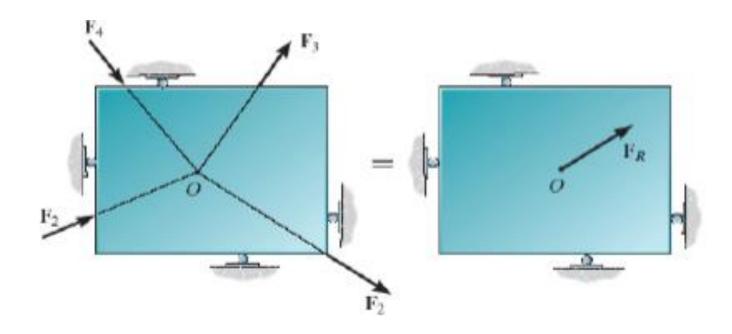
$$(F_R)_x = \Sigma F_x$$

 $(F_R)_y = \Sigma F_y$
 $(M_R)_O = \Sigma M_O + \Sigma M$



Simplification of a force and couple system

Concurrent Force system
 A concurrent force system is where lines of action of all the forces intersect at a common point O



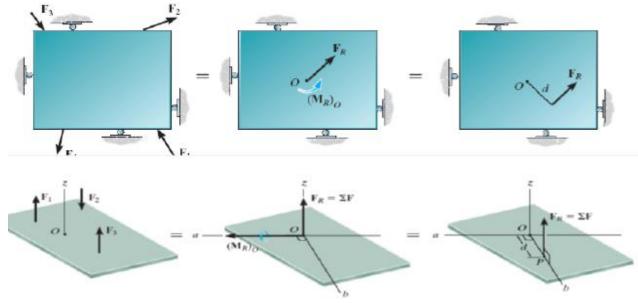
Simplification of a force and couple system

Coplanar Force system
 Lines of action of all the forces lie in the same plane
 Resultant force of this system also lies in this plane



$$F_R = \Sigma F$$

 $(M_R)_O = F_R d = \Sigma M_O \text{ or } d = \Sigma M_O / F_R$

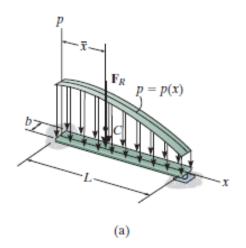


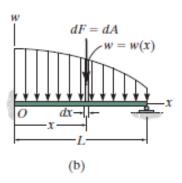
Reduction of a simple distributed loading

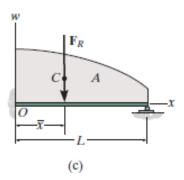
Large surface area of a body may be subjected to distributed loadings, often defined as pressure measured in Pascal (Pa):

$$1 \text{ Pa} = 1 \text{N/m}^2$$

■ Loading Along a single axis: for a constant width b, w(x)=p(x) b (N/m). This coplanar parallel force system can be replaced with an equivalent resultant force F_R acting at a specific location on the beam.







Reduction of a simple distributed loading

■ Magnitude of resultant force: dF=w(x) dx=dA, with $F_R=\Sigma F \Rightarrow$

$$F_R = \int_L w(x)dx = \int_A dA = A$$

Location of the resultant force:

$$M_{RO} = \Sigma M_O \Rightarrow -\bar{x}F_R = -\int_L xw(x)dx$$

The location \bar{x} of the line of action of F_R ,

$$\overline{x} = \frac{\int_{L} xw(x)dx}{\int_{L} w(x)dx} = \frac{\int_{A} xdA}{\int_{A} dA}$$

The resultant force has a line of action which passes through the centroid C (geometric center) of the area under the loading diagram.