

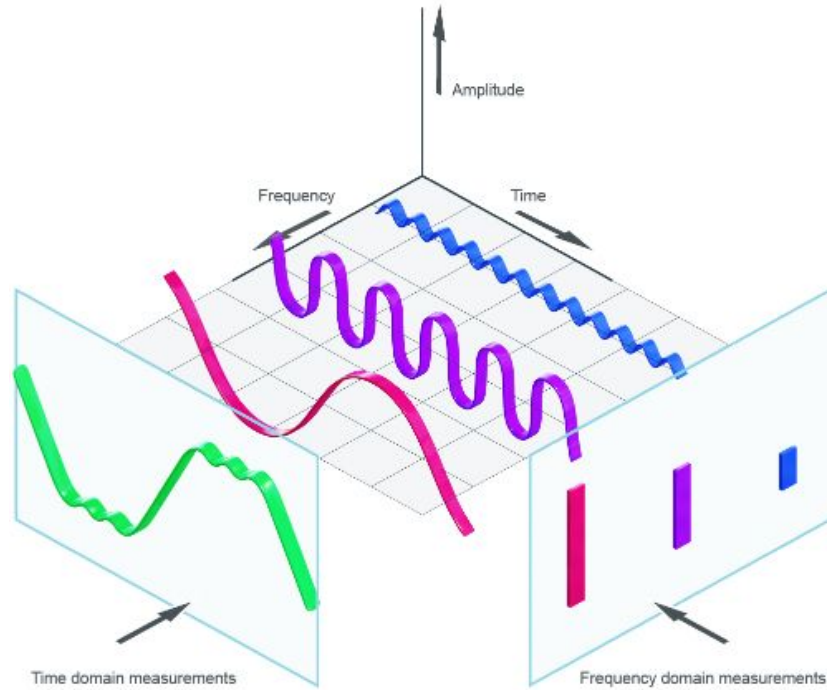
# Introduction to Robotics

## CSE 461

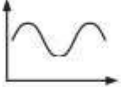


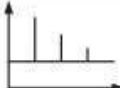


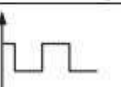
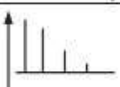


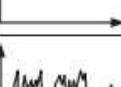





### Lecture 11: Introduction to Control System Theory (Part 2)

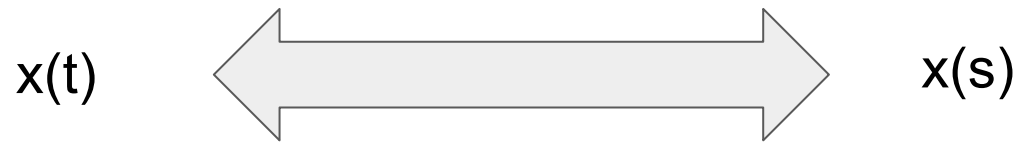
Niloy Irtisam  
Lecturer, Dept. of Computer Science and Engineering  
Brac University

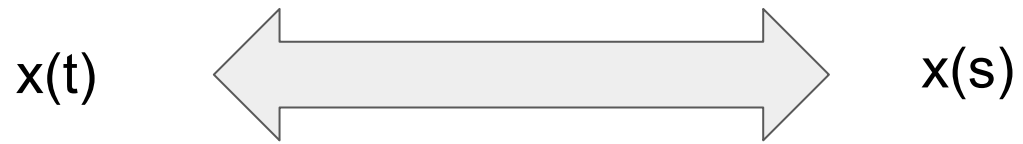
# Time Domain to Frequency Domain



# Signals




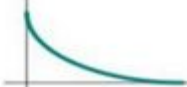


Waveform	Time domain	Frequency domain
Sinewave		
Triangle		
Sawtooth		
Rectangle		
Pulse		
Random noise		
Bandlimited noise		
Random binary sequence		



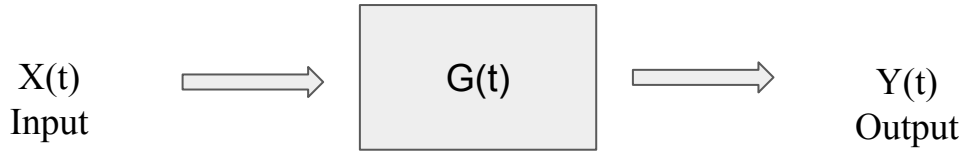


Laplace Transform

# Common Laplace Transform

Name	$f(t)$		$F(s)$
Impulse $\delta$	$f(t) = \begin{cases} 1 & t = 0 \\ 0 & t > 0 \end{cases}$		1
Step	$f(t) = 1$		$\frac{1}{s}$
Ramp	$f(t) = t$		$\frac{1}{s^2}$
Exponential	$f(t) = e^{-at}$		$\frac{1}{s+a}$
Sine	$f(t) = \sin(\omega t)$		$\frac{\omega}{s^2 + \omega^2}$
Damped Sine	$f(t) = e^{-at} \sin(\omega t)$		$\frac{\omega}{(s+a)^2 + \omega^2}$

# Block Diagram

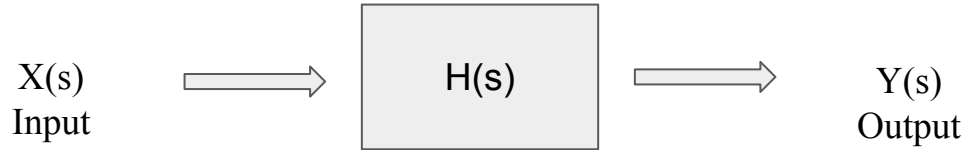


$$Y(t) = X(t) * G(t)$$

$$Y(t)/X(t) = G(t)$$

$$\text{Gain} = Y(t)/X(t) = \text{Output/Input}$$

# Transfer Function



- Definition: Transfer function is defined as the ratio of LT of output to the L.T of input. When all the initial condition assume to be zero.

$$H(s) = Y(s) / X(s)$$

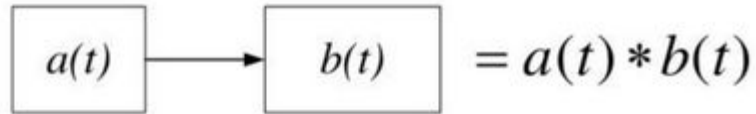
- Relates the output of a linear system to its input.
- Describes how a linear system responds to an impulse, called impulse response



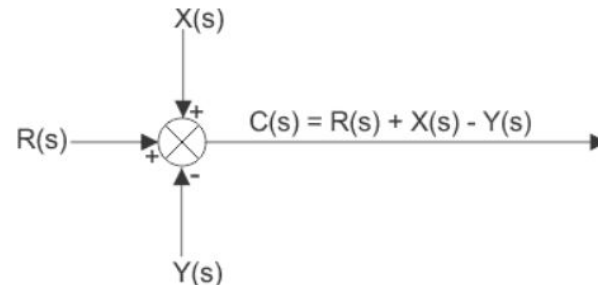
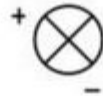
# Block Diagram

- Rules

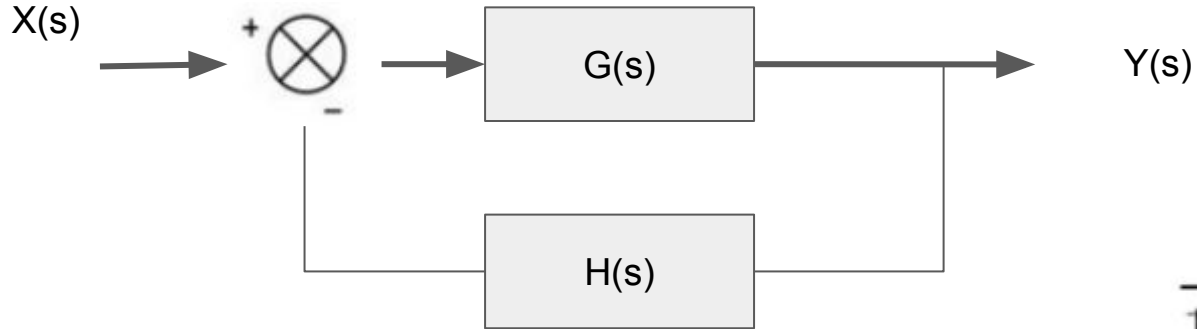
- Cascaded elements: convolution



- Summation and deference elements

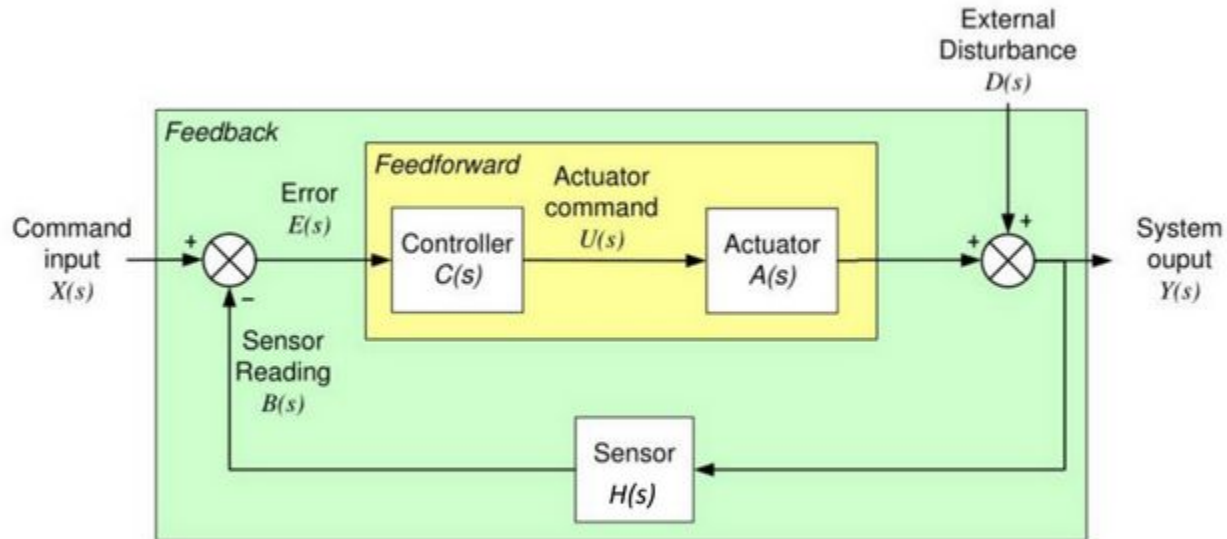


- Feedback Connection



$$\frac{G(s)}{1 \mp G(s)H(s)} * X(s)$$

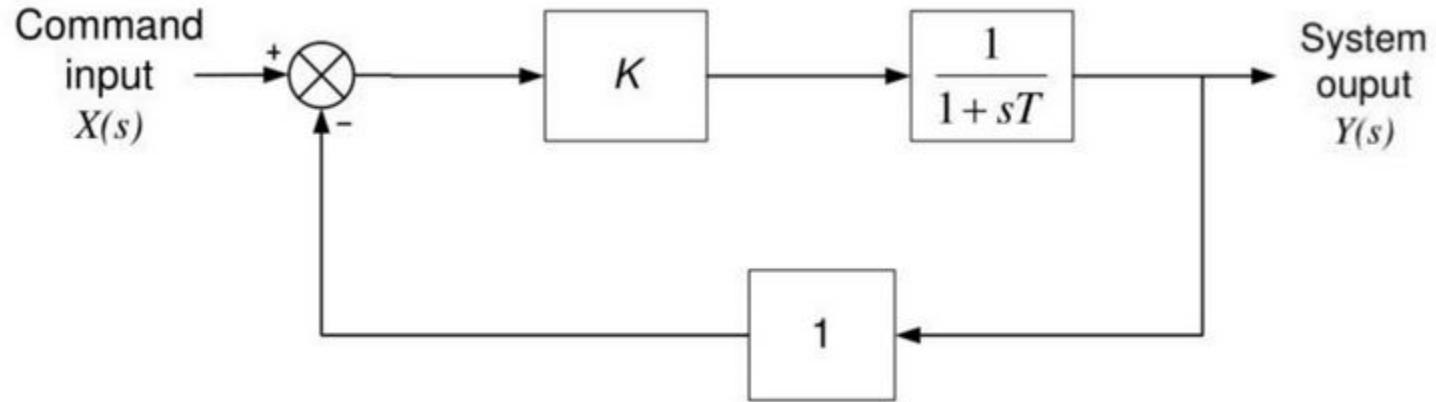
# Key Transfer Function



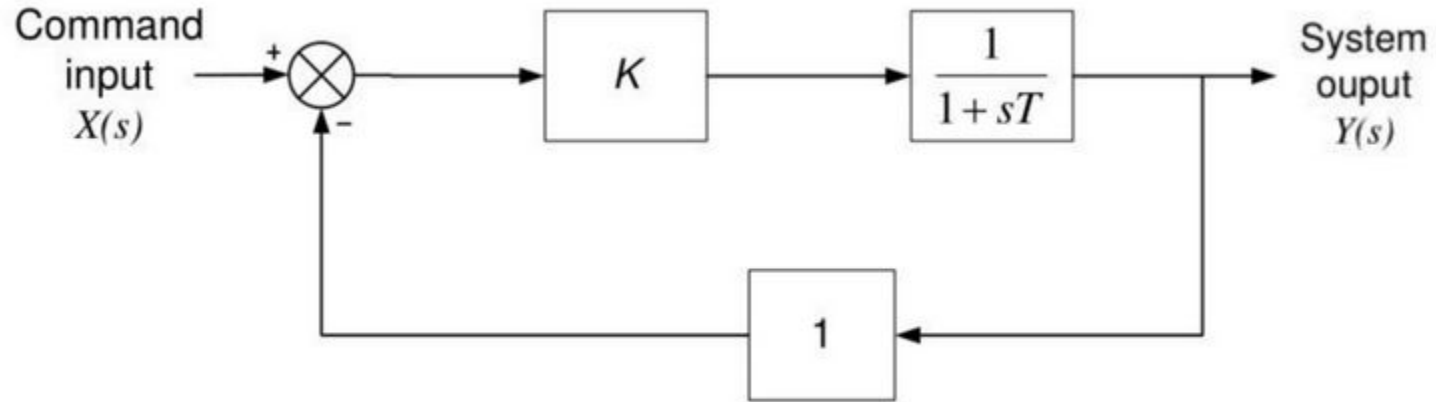
$$\text{Feedforward: } \frac{Y(s)}{E(s)} = C(s)A(s)$$

$$\text{Feedback: } \frac{Y(s)}{X(s)} = \frac{C(s)A(s)}{1 + C(s)A(s)H(s)}$$

# Transfer Function



# Transfer Function

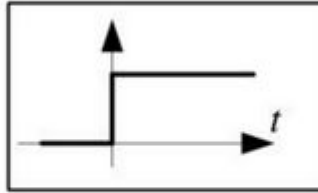


$$\frac{Y(s)}{X(s)} = \frac{K}{1 + K + sT}$$

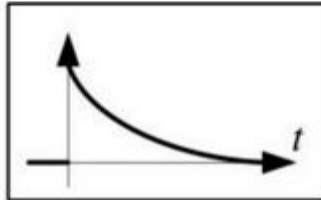
# Steady State Vs Transient

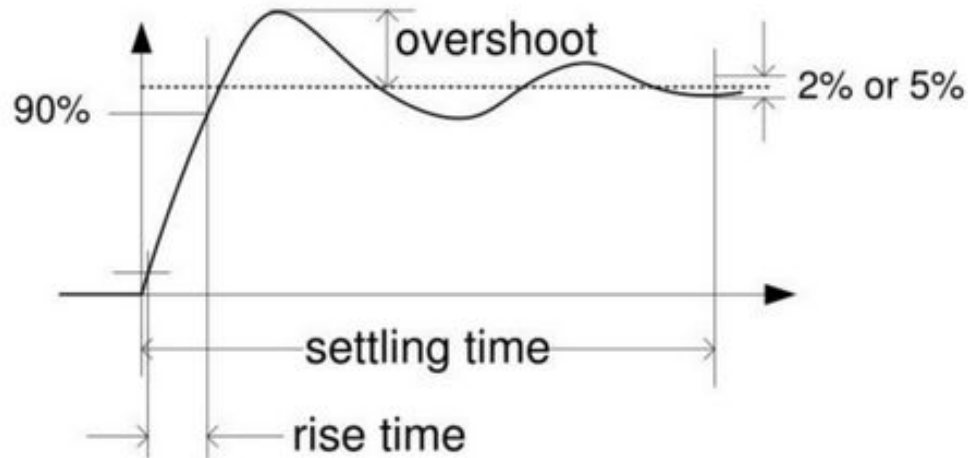
- Step Response illustrates how a system response can be decomposed into two components

- Steady-state part:



- Transient





**overshoot** -- % of final value exceeded at first *oscillation*

**rise time** -- time to span from 10% to 90% of the final value

**settling time** -- time to reach within 2% or 5% of the final value

# PID Controller

Proportional control :  $u(t) = K_p e(t)$   $\frac{U(s)}{E(s)} = K_p$

Integral control :  $u(t) = K_i \int_0^t e(t) dt$   $\frac{U(s)}{E(s)} = \frac{K_i}{s}$

Differential control :  $u(t) = K_d \frac{d}{dt} e(t)$   $\frac{U(s)}{E(s)} = K_d s$



- It produces an output, which is the combination of the outputs of proportional , integral & derivative controllers

$$u(t) \propto e(t) + \int e(t) + \frac{d}{dt} e(t)$$

$$\gg u(t) = K_P e(t) + K_I \int e(t) + K_D \frac{d}{dt} e(t)$$

Laplace transform in both side

$$U(S) = K_P E(S) + \frac{K_I}{S} E(S) + K_D S E(S)$$

$$U(S) = E(S) \left( K_P + \frac{K_I}{S} + K_D S \right)$$

$$\frac{U(S)}{E(S)} = \left( K_P + \frac{K_I}{S} + K_D S \right) = \frac{K_P S + K_I + K_D S^2}{S}$$

# Effect of Controller Functions

- Proportional Action

Simplest Controller Function, The P term helps to reduce the steady-state error and improve the system's responsiveness. However, too high of a proportional gain can lead to instability and oscillations in the system's response.

- Integral Action

Eliminates steady-state error, The I term helps to improve the system's stability and eliminate any bias in the system. However, too high of an integral gain can lead to overshoot and instability in the system's response.

- Derivative Action (“rate control”)

Effective in transient periods, The D term helps to improve the system's stability and reduce the effects of disturbances in the system. However, too high of a derivative gain can lead to noise amplification and instability in the system's response.

## How to get the PID parameter values ?

- If we know the transfer function, analytical methods can be used (e.g., root-locus method) to meet the transient and steady-state specs.
- When the system dynamics are not precisely known, we must resort to experimental approaches.

## Ziegler-Nichols Rules for Tuning PID Controller

Using only Proportional control, turn up the gain until the system oscillates without dying down, i.e., is marginally stable. Assume that  $K$  and  $P$  are the resulting gain and oscillation period, respectively.

Then Use

for P control

$$K_p = 0.5 K$$

for PI control

$$K_p = 0.45 K$$

$$K_i = 1.2 / P$$

for PID control

$$K_p = 0.6 K$$

$$K_i = 2.0 / P$$

$$K_d = P / 8.0$$

Ziegler-Nichols Tuning  
for second or higher  
order systems

- PID control---most widely used control strategy today
- Over 90% of control loops employ PID control, often the derivative gain set to zero (PI control)
- The three terms are intuitive---a non - specialist can grasp the essentials of the PID controller's action. It does not require the operator to be familiar with advanced math to use PID controllers
- Engineers prefer PID controls over untested solutions

# Next Class

Machine Learning