

# ME232: Dynamics

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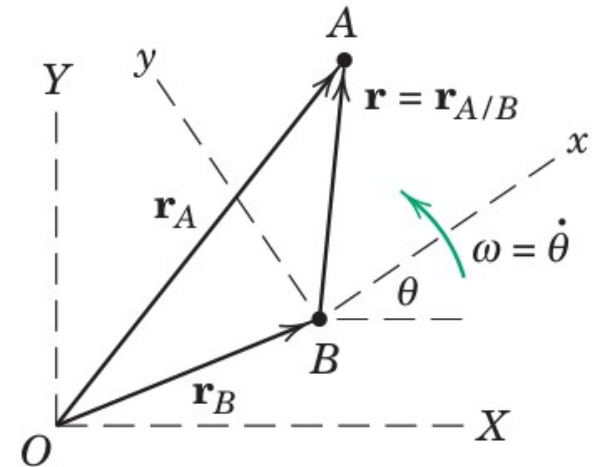
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Room # 106

# Motion relative to rotating axes

- Till now in our discussion we have used non-rotating reference axes to describe relative velocity and relative acceleration.
- Use of rotating reference axes greatly facilitates the solution of many problems in kinematics where motion is generated **within a system** or **observed from a system which itself is rotating**.
- An example of such a motion is the movement of a fluid particle along the curved vane of a centrifugal pump, where the **path relative to the vanes** of the impeller becomes an important design consideration.

Consider the plane motion of two particles  $A$  and  $B$  in the fixed  $X$ - $Y$  plane. For the time being, we will consider  $A$  and  $B$  to be moving independently of one another for the sake of generality. We observe the motion of  $A$  from a moving reference frame  $x$ - $y$  which has its origin attached to  $B$  and which rotates with an angular velocity  $\omega = \dot{\theta}$ .



We may write this angular velocity as the vector  $\omega = \omega \mathbf{k}$  (direction of the vector is established by the right-hand rule). The absolute position vector of  $A$  is given,

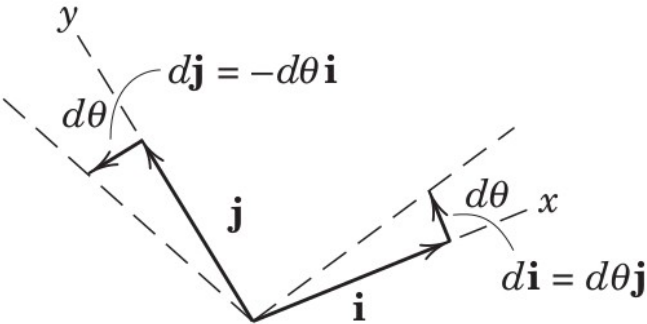
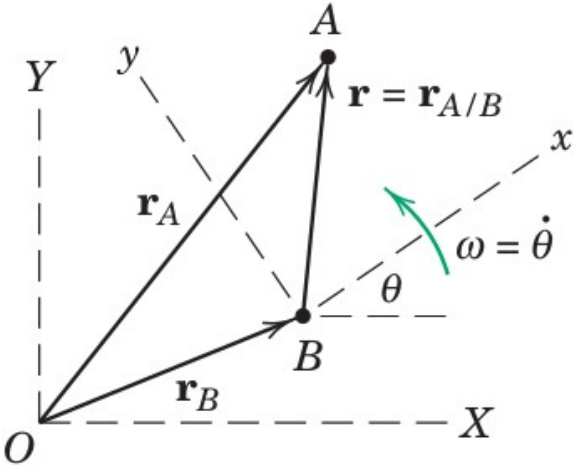
$$\mathbf{r}_A = \mathbf{r}_B + \mathbf{r}_{A/B} = \mathbf{r}_B + (x \mathbf{i} + y \mathbf{j}), \qquad \dots\dots\dots(9)$$

where  $\mathbf{i}$  and  $\mathbf{j}$  are unit vectors attached to the  $x$ - $y$  frame.

To obtain the velocity and acceleration equations we successively differentiate (9) w.r.t. time. The unit vectors  $\mathbf{i}$  and  $\mathbf{j}$  are now rotating with the  $x$ - $y$  axes and, therefore, have time derivatives. These derivatives may be seen from the figure, which shows the infinitesimal change in each unit vector during time  $dt$  as the reference axes rotate through an angle  $d\theta = \omega \, dt$ .

The differential change in  $\mathbf{i}$  is  $d\mathbf{i} = d\theta \, \mathbf{j}$ , and the differential change in  $\mathbf{j}$  is  $d\mathbf{j} = -d\theta \, \mathbf{i}$ . Thus,

$$\dot{\mathbf{i}} = \omega \mathbf{j} = \boldsymbol{\omega} \times \mathbf{i}, \text{ and } \dot{\mathbf{j}} = -\omega \mathbf{i} = \boldsymbol{\omega} \times \mathbf{j}. \qquad \dots\dots\dots(10)$$



*Relative velocity:*

Expression for relative velocity can be obtained by differentiating (10) w.r.t. time as,

$$\begin{aligned}\dot{\boldsymbol{r}}_A &= \dot{\boldsymbol{r}}_B + \dot{\boldsymbol{r}}_{A/B} = \dot{\boldsymbol{r}}_B + \left[ (\dot{x}\boldsymbol{i} + \dot{y}\boldsymbol{j}) + (x\dot{\boldsymbol{i}} + y\dot{\boldsymbol{j}}) \right] \\ \boldsymbol{v}_A &= \boldsymbol{v}_B + [\boldsymbol{v}_{\text{rel}} + (x\boldsymbol{\omega} \times \boldsymbol{i} + y\boldsymbol{\omega} \times \boldsymbol{j})] \\ \boldsymbol{v}_A &= \boldsymbol{v}_B + \boldsymbol{v}_{\text{rel}} + \boldsymbol{\omega} \times (x\boldsymbol{i} + y\boldsymbol{j}) \\ \boldsymbol{v}_A &= \boldsymbol{v}_B + \boldsymbol{v}_{\text{rel}} + \boldsymbol{\omega} \times \boldsymbol{r} \qquad \qquad \qquad \dots\dots\dots(11)\end{aligned}$$

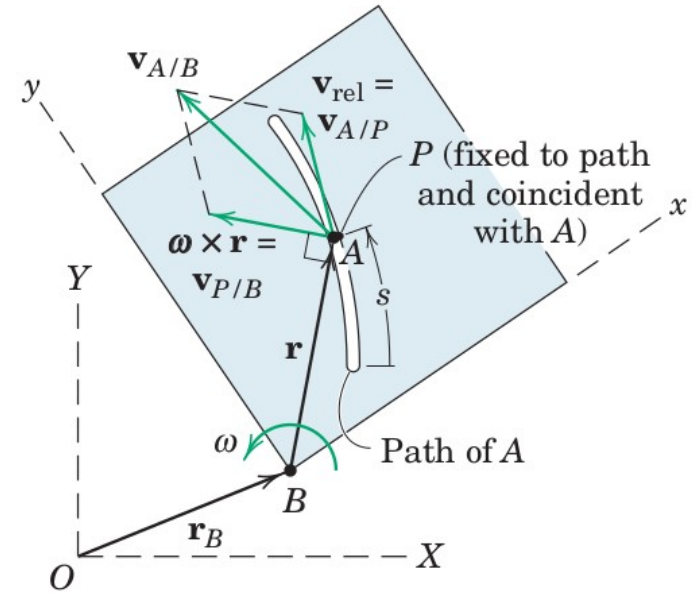
Comparing (11) with the expression for non-rotating reference axes it is observed that  $\boldsymbol{v}_{A/B} = \boldsymbol{\omega} \times \boldsymbol{r} + \boldsymbol{v}_{\text{rel}}$ , from which it can be concluded that **the term  $\boldsymbol{\omega} \times \boldsymbol{r}$  is the difference between the relative velocities as measured from non-rotating and rotating axes.**

To visualize the meaning of the last two terms of (11), we observe the motion of particle  $A$  in a curved slot in a plate which represents the rotating  $x$ - $y$  reference system.

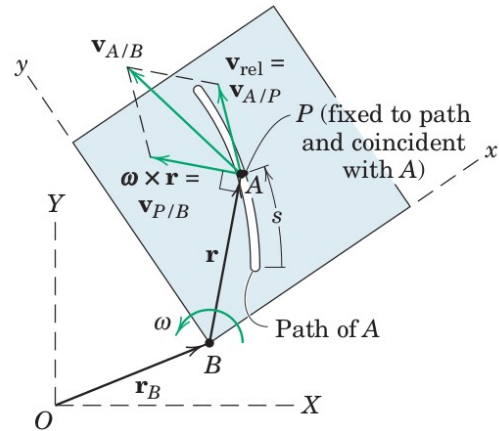
The velocity of  $A$  as measured relative to the plate,  $\mathbf{v}_{\text{rel}}$ , would be tangent to the path fixed in the  $x$ - $y$  plate and would have a magnitude  $\dot{s}$ , where  $s$  is measured along the path.

This relative velocity may also be viewed as the velocity  $\mathbf{v}_{A/P}$  **relative to a point  $P$  attached to the plate and coincident with  $A$  at the instant under consideration.**

The term  $\boldsymbol{\omega} \times \mathbf{r}$  has a magnitude  $r\dot{\theta}$  and a direction normal to  $\mathbf{r}$  and is the **velocity of point  $P$  relative to  $B$  as seen from nonrotating axes attached to  $B$ .**



The following comparison will help establish the equivalence of, and clarify the differences between, the relative-velocity equations written for rotating and nonrotating reference axes:



$$\begin{aligned}
 \boldsymbol{v}_A &= \boldsymbol{v}_B + \boldsymbol{\omega} \times \boldsymbol{r} + \boldsymbol{v}_{\text{rel}} \\
 \boldsymbol{v}_A &= \boldsymbol{v}_B + \boldsymbol{v}_{P/B} + \boldsymbol{v}_{A/P} \\
 \boldsymbol{v}_A &= \boldsymbol{v}_P + \boldsymbol{v}_{A/P} \\
 \boldsymbol{v}_A &= \boldsymbol{v}_B + \boldsymbol{v}_{A/B}
 \end{aligned}
 \tag{11a}$$

In the second equation, the term  $\boldsymbol{v}_{P/B}$  is measured from a nonrotating position. The term  $\boldsymbol{v}_{A/P}$  is the same as  $\boldsymbol{v}_{\text{rel}}$  and is the velocity of  $A$  as measured in the  $x$ - $y$  frame. In the third equation,  $\boldsymbol{v}_P$  is the absolute velocity of  $P$  and represents the effect of the moving coordinate system, both transnational and rotational. The fourth equation is the same as that developed for nonrotating axes, and it is seen that

$$\boldsymbol{v}_{A/B} = \boldsymbol{v}_{P/B} + \boldsymbol{v}_{A/P} = \boldsymbol{\omega} \times \boldsymbol{r} + \boldsymbol{v}_{\text{rel}}.$$

31

# Transformation of a time derivative

(11) represents a transformation of the time derivative of the position vector between rotating and nonrotating axes. This result can be generalized to apply to the time derivative of any vector quantity  $\mathbf{V} = V_x \mathbf{i} + V_y \mathbf{j}$ . Accordingly, the total time derivative with respect to the  $X$ - $Y$  system is

$$\begin{aligned} \left(\frac{d\mathbf{V}}{dt}\right)_{XY} &= \left(\dot{V}_x \mathbf{i} + \dot{V}_y \mathbf{j}\right) + \left(V_x \dot{\mathbf{i}} + V_y \dot{\mathbf{j}}\right) \\ \left(\frac{d\mathbf{V}}{dt}\right)_{XY} &= \left(\frac{d\mathbf{V}}{dt}\right)_{xy} + \boldsymbol{\omega} \times (V_x \mathbf{i} + V_y \mathbf{j}) \\ \left(\frac{d\mathbf{V}}{dt}\right)_{XY} &= \left(\frac{d\mathbf{V}}{dt}\right)_{xy} + \boldsymbol{\omega} \times \mathbf{V} \qquad \dots\dots\dots(12) \end{aligned}$$

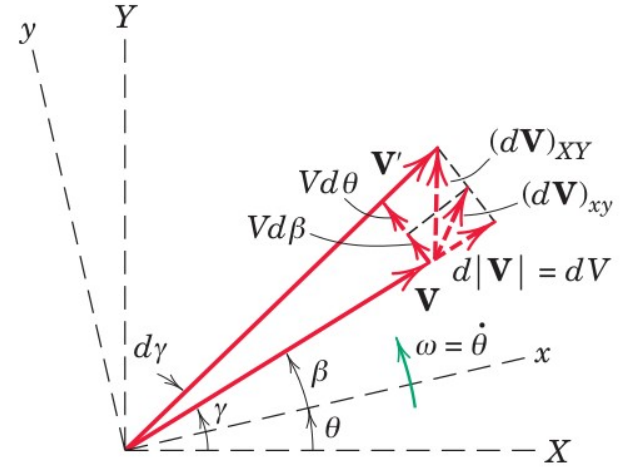
The first term represents **the part of the total derivative of  $V$  which is measured relative to the  $x$ - $y$  reference system**, and the second term represents **the part of the derivative due to the rotation of the reference system**.

To understand the physical significance of (12) consider a vector  $\mathbf{V}$  at time  $t$ , observed both in the fixed axes  $X$ - $Y$  and in the rotating axes  $x$ - $y$ . Because we are dealing with the effects of rotation only, the vector may be drawn through the coordinate origin without loss of generality. During time  $dt$ , the vector swings to position  $\mathbf{V}'$ , and the observer in  $x$ - $y$  measures the two components

- (a)  $dV$  due to its change in magnitude, and
- (b)  $Vd\beta$  due to its rotation  $d\beta$  relative to  $x$ - $y$ .

Hence, the derivative  $(d\mathbf{V}/dt)_{xy}$  which the rotating observer measures has the components  $dV/dt$  and  $Vd\beta/dt = V\dot{\beta}$ .

The remaining part of the total time derivative not measured by the rotating observer has the magnitude  $Vd\theta/dt$  and, expressed as a vector, is  $\boldsymbol{\omega} \times \mathbf{V}$ . Thus, we see from the diagram that



$$(\dot{\mathbf{V}})_{XY} = (\dot{\mathbf{V}})_{xy} + \boldsymbol{\omega} \times \mathbf{V}.$$



# Relative acceleration:

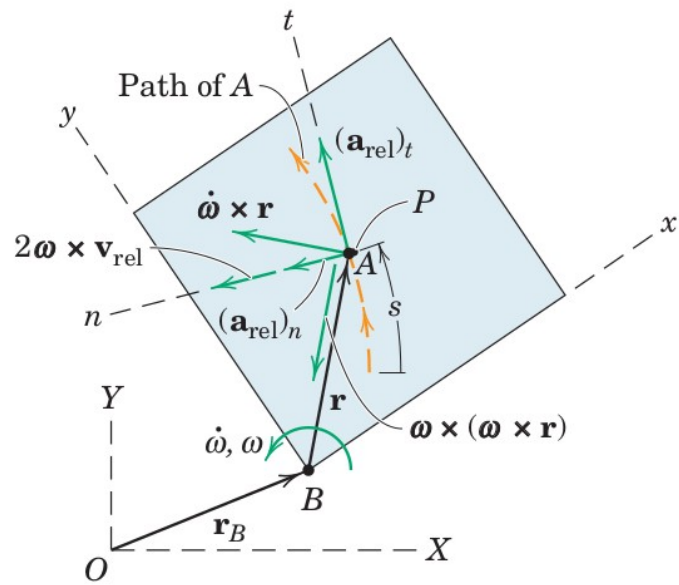
Expression for relative acceleration can be obtained by differentiating (11) w.r.t. time as,

$$\begin{aligned}\dot{\mathbf{v}}_A &= \dot{\mathbf{v}}_B + \dot{\mathbf{v}}_{\text{rel}} + \dot{\boldsymbol{\omega}} \times \mathbf{r} + \boldsymbol{\omega} \times \dot{\mathbf{r}} \\ \mathbf{a}_A &= \mathbf{a}_B + (\mathbf{a}_{\text{rel}} + \boldsymbol{\omega} \times \mathbf{v}_{\text{rel}}) + \dot{\boldsymbol{\omega}} \times \mathbf{r} + \boldsymbol{\omega} \times (\mathbf{v}_{\text{rel}} + \boldsymbol{\omega} \times \mathbf{r}) \quad (\text{from 12}) \\ \mathbf{a}_A &= \mathbf{a}_B + \dot{\boldsymbol{\omega}} \times \mathbf{r} + \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r}) + 2\boldsymbol{\omega} \times \mathbf{v}_{\text{rel}} + \mathbf{a}_{\text{rel}} \quad \dots\dots\dots(13)\end{aligned}$$

Equation (13) is the general vector expression for the absolute acceleration of a particle  $A$  in terms of its acceleration  $\mathbf{a}_{\text{rel}}$  measured relative to a moving coordinate system which rotates with an angular velocity  $\boldsymbol{\omega}$  and an angular acceleration  $\dot{\boldsymbol{\omega}}$ .

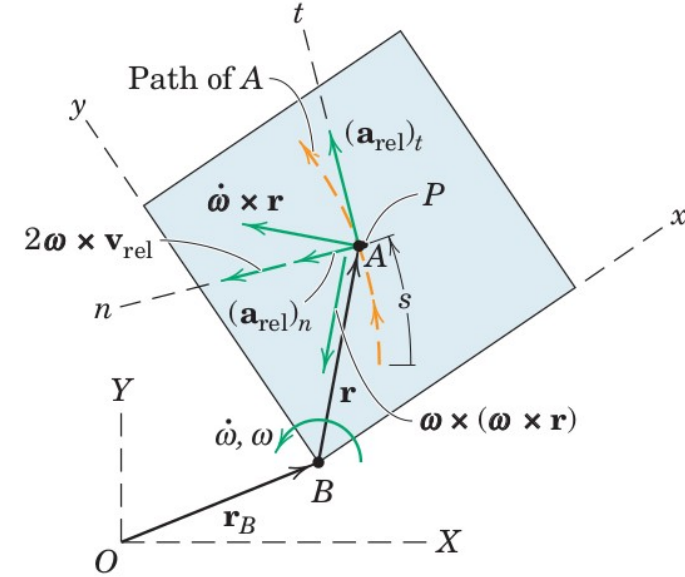
The terms  $\dot{\boldsymbol{\omega}} \times \mathbf{r}$  and  $\boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r})$  are shown.

They represent, respectively, the tangential and normal components of the acceleration  $\mathbf{a}_{P/B}$  of the coincident point  $P$  in its circular motion with respect to  $B$ .



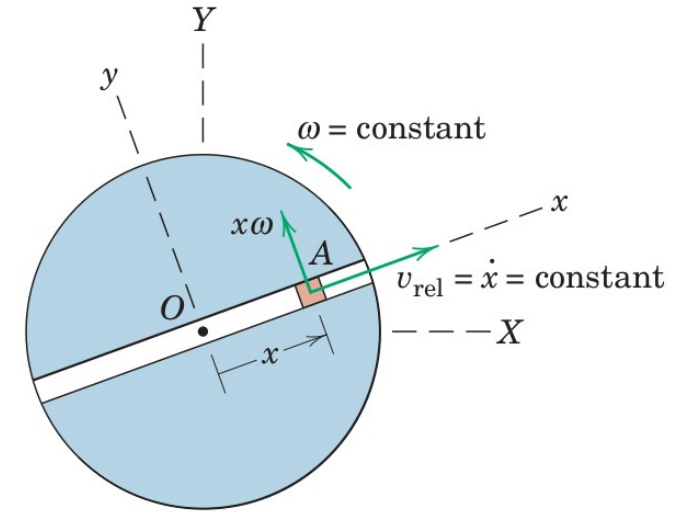
This motion would be observed from a set of nonrotating axes moving with  $B$ . The magnitude of  $\dot{\boldsymbol{\omega}} \times \mathbf{r}$  is  $r\ddot{\theta}$  and its direction is tangent to the circle. The magnitude of  $\boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r})$  is  $r\omega^2$  and its direction is from  $P$  to  $B$  along the normal to the circle.

The acceleration of  $A$  relative to the plate along the path,  $\mathbf{a}_{\text{rel}}$ , may be expressed in rectangular, normal and tangential, or polar coordinates in the rotating system. Frequently,  $n$ - and  $t$ -components are used, and these components are shown in figure. The tangential component has the magnitude  $(a_{\text{rel}})_t = \ddot{s}$ , where  $s$  is the distance measured along the path to  $A$ . The normal component has the magnitude  $(a_{\text{rel}})_n = v_{\text{rel}}^2/\rho$ , where  $\rho$  is the radius of curvature of the path as measured in  $x$ - $y$ . The sense of this vector is always toward the center of curvature.



The term  $2\boldsymbol{\omega} \times \mathbf{v}_{\text{rel}}$ , is called the **Coriolis acceleration**. It represents the **difference between the acceleration of  $A$  relative to  $P$  as measured from nonrotating axes and from rotating axes**. The direction is always normal to the vector  $\mathbf{v}_{\text{rel}}$ , and the sense is established by the right-hand rule for the cross product.

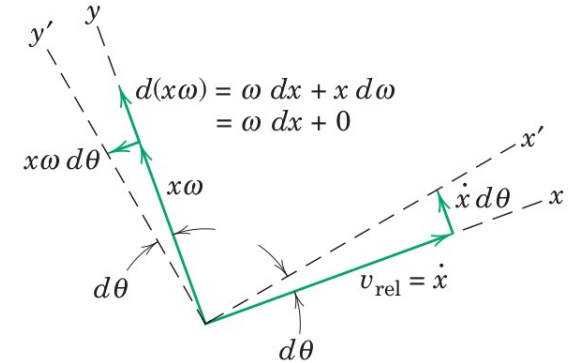
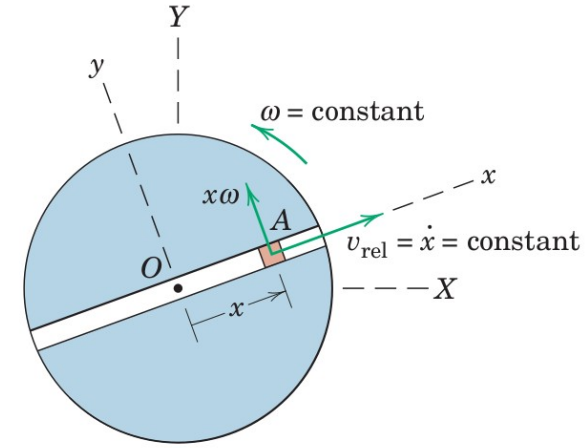
The Coriolis acceleration  $\mathbf{a}_{\text{Cor}} = 2\boldsymbol{\omega} \times \mathbf{v}_{\text{rel}}$  is difficult to visualize because it is composed of two separate physical effects. To help with this visualization, we will consider the simplest possible motion in which this term appears. Consider a rotating disk with a radial slot in which a small particle  $A$  is confined to slide. Let the disk turn with a constant angular velocity  $\boldsymbol{\omega}$  and let the particle move along the slot with a constant speed  $\mathbf{v}_{\text{rel}} = \dot{\mathbf{x}}$  relative to the slot. The velocity of  $A$  has the two components (a)  $\dot{\mathbf{x}}$  due to motion along the slot, and (b)  $\mathbf{x}\boldsymbol{\omega}$  due to the rotation of the slot.



The changes in these two velocity components due to the rotation of the disk are shown for the interval  $dt$ , during which the  $x$ - $y$  axes rotate with the disk through the angle  $d\theta$  to  $x'$ - $y'$ .

The velocity increment due to the change in direction of  $\mathbf{v}_{\text{rel}}$  is  $\dot{x} d\theta$  and that due to the change in magnitude of  $x\omega$  is  $\omega dx$ , both being in the  $y$ -direction normal to the slot. Dividing each increment by  $dt$  and adding give the sum  $\omega \dot{x} + \dot{x} \omega = 2\dot{x}\omega$ , which is the magnitude of the Coriolis acceleration  $2\boldsymbol{\omega} \times \mathbf{v}_{\text{rel}}$ .

Dividing the remaining velocity increment  $x\omega d\theta$  due to the change in direction of  $x\omega$  by  $dt$  gives  $x\omega \dot{\theta} = x\omega^2$ , which is the acceleration of a point  $P$  fixed to the slot and momentarily coincident with the particle  $A$ .



Now applying (13) for the same motion, we see that

$$\mathbf{a}_A = \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r}) + 2\boldsymbol{\omega} \times \mathbf{v}_{\text{rel}}$$

Replacing  $\mathbf{r}$  by  $x\mathbf{i}$ ,  $\boldsymbol{\omega}$  by  $\omega\mathbf{k}$ , and  $\mathbf{v}_{\text{rel}}$  by  $\dot{x}\mathbf{i}$  gives

$$\mathbf{a}_A = \omega^2 x\mathbf{i} + 2\omega\dot{x}\mathbf{j}$$

# Motion difference between two instantaneously coincident points:

$$OP_3 = OP_2 + P_2P_3$$

$$v_{P_3} = v_{P_2} + \omega \times P_3P_2 + v_{P_3/P_2}$$

$$a_{P_3} = a_{P_2} + \dot{\omega} \times P_3P_2 + \omega \times (\omega \times P_3P_2) + 2\omega \times v_{P_3/P_2} + a_{P_3/P_2}$$

Substitute  $P_2P_3=0$  now,

$$v_{P_3} = v_{P_2} + v_{P_3/P_2}$$

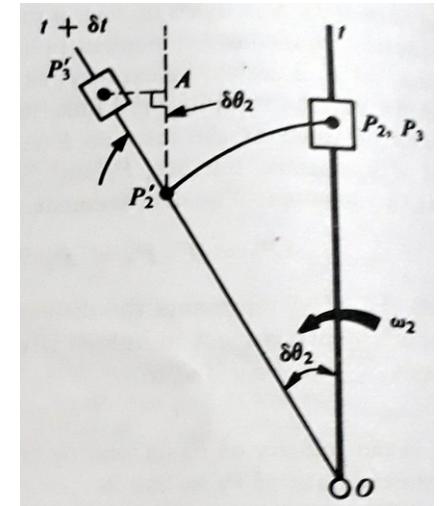
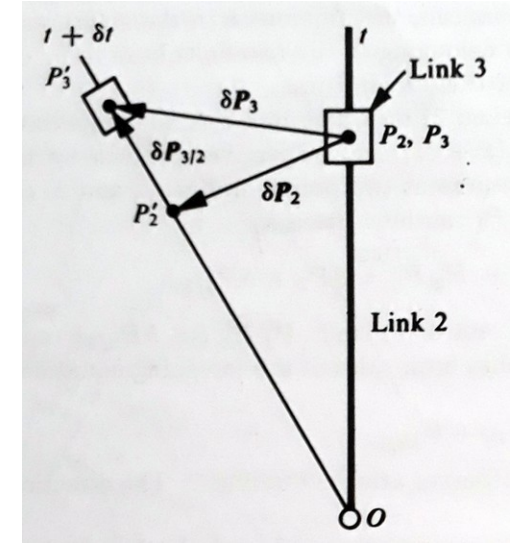
$$a_{P_3} = a_{P_2} + 2\omega \times v_{P_3/P_2} + a_{P_3/P_2}$$

The Coriolis component of acceleration can be understood as follows:

$$AP'_3 = P'_2P'_3 \cdot \delta\theta_2 = V_{P_3/P_2}\delta t \cdot \omega_2\delta t = V_{P_3/P_2} \cdot \omega_2\delta t^2$$

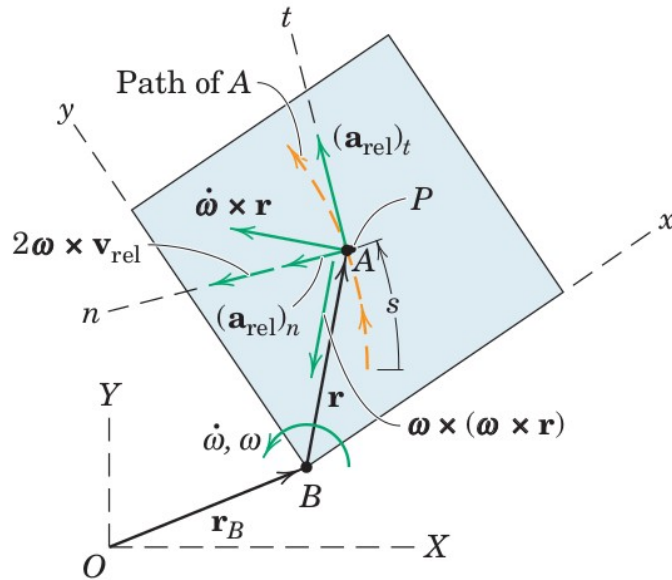
This displacement term is proportional to the square of time elapsed. Therefore this displacement must be due to an additional acceleration of  $P_3$  in transverse direction. If the magnitude of this additional acceleration is  $a_c$ , then

$$\frac{1}{2}a_c\delta t^2 = V_{P_3/P_2} \cdot \omega_2 \cdot \delta t^2 \Rightarrow a_c = 2V_{P_3/P_2}\omega_2.$$



## Rotating vs. Nonrotating system

The following comparison will help to establish the equivalence of, and clarify the differences between, the relative-acceleration equations written for rotating and nonrotating reference axes:



$$\mathbf{a}_A = \mathbf{a}_B + \dot{\boldsymbol{\omega}} \times \mathbf{r} + \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r}) + 2\boldsymbol{\omega} \times \mathbf{v}_{\text{rel}} + \mathbf{a}_{\text{rel}}$$

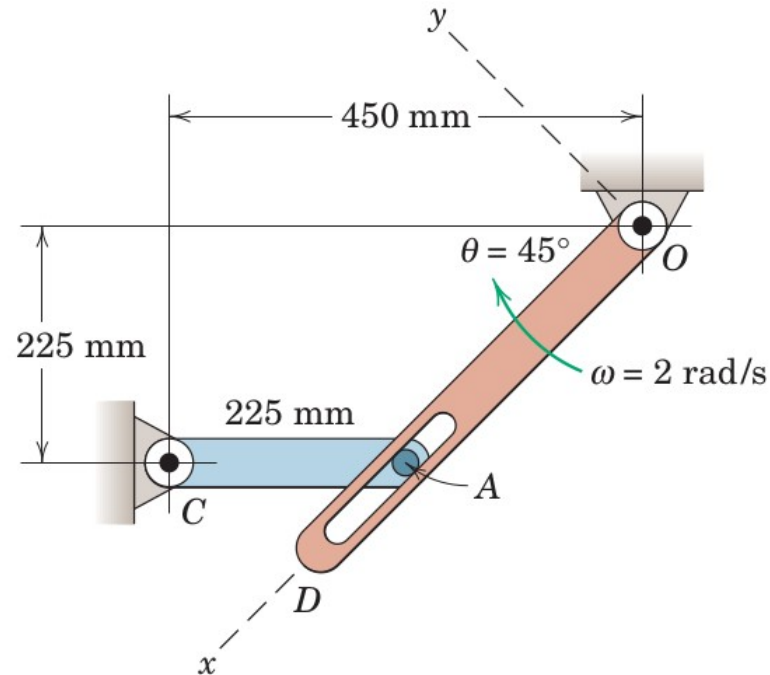
$$\mathbf{a}_A = \mathbf{a}_B + \mathbf{a}_{P/B} + \mathbf{a}_{A/P}$$

$$\mathbf{a}_A = \mathbf{a}_P + \mathbf{a}_{A/P}$$

$$\mathbf{a}_A = \mathbf{a}_B + \mathbf{a}_{A/B}$$

## Example 9

The pin  $A$  of the hinged link  $AC$  is confined to move in the rotating slot of link  $OD$ . The angular velocity of  $OD$  is  $\omega = 2 \text{ rad/s}$  clockwise and is constant for the interval of motion concerned. For the position where  $\theta = 45^\circ$  with  $AC$  horizontal, determine (a) the velocity of pin  $A$  and the velocity of  $A$  relative to the rotating slot in  $OD$ , (b) determine the angular acceleration of  $AC$  and the acceleration of  $A$  relative to the rotating slot in arm  $OD$ .





Pin  $A$  always moves along the slot, hence we use a rotating coordinate system  $x$ - $y$  attached to arm  $OD$ . The expression of velocity of  $A$ , with the origin at the fixed point  $O$  as,

$$\mathbf{v}_A = \boldsymbol{\omega}_{OD} \times \mathbf{r} + \mathbf{v}_{\text{rel}}.$$

Point  $A$  also moves in a circular path about  $C$ , hence

$$\mathbf{v}_A = \boldsymbol{\omega}_{CA} \times \mathbf{r}_{CA} = \omega_{CA} \mathbf{k} \times (225/\sqrt{2})(-\mathbf{i} - \mathbf{j}) = (225/\sqrt{2})\omega_{CA} (\mathbf{i} - \mathbf{j})$$

where  $\omega_{CA}$  is assumed to be clockwise arbitrarily.

The vector from the origin to the point  $P$  on  $OD$  coincident with  $A$  is

$$\mathbf{r} = OP \mathbf{i} = 225\sqrt{2} \mathbf{i} \text{ mm.}$$

Thus,  $\boldsymbol{\omega}_{OD} \times \mathbf{r} = 450\sqrt{2} \mathbf{j} \text{ mm/s.}$

Finally, the relative-velocity term  $\mathbf{v}_{\text{rel}}$  is the velocity measured by an observer attached to the rotating reference frame and is  $\mathbf{v}_{\text{rel}} = \dot{\mathbf{x}} \mathbf{i}$ .

Substituting into the relative-velocity equation and solving it gives,  $\omega_{CA} = -4 \text{ rad/s}$  and  $\dot{\mathbf{x}} = \mathbf{v}_{\text{rel}} = -450\sqrt{2} \text{ mm/s.}$  Now  $\mathbf{v}_A$  can also be calculated.

The expression for acceleration of point  $A$  with the origin at the fixed point  $O$ ,

$$\mathbf{a}_A = \dot{\boldsymbol{\omega}} \times \mathbf{r} + \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r}) + 2\boldsymbol{\omega} \times \mathbf{v}_{\text{rel}} + \mathbf{a}_{\text{rel}}$$

Acceleration of point  $A$  can also be written as,

$$\mathbf{a}_A = \dot{\boldsymbol{\omega}}_{CA} \times \mathbf{r}_{CA} + \boldsymbol{\omega}_{CA} \times (\boldsymbol{\omega}_{CA} \times \mathbf{r}_{CA})$$

Substituting these values from solution of (a),

$$\mathbf{a}_A = \dot{\omega}_{CA} \mathbf{k} \times \frac{225}{\sqrt{2}}(-\mathbf{i} - \mathbf{j}) - 4\mathbf{k} \times \left( -4\mathbf{k} \times \frac{225}{\sqrt{2}}[-\mathbf{i} - \mathbf{j}] \right)$$

As  $\boldsymbol{\omega}$  is constant,  $\dot{\boldsymbol{\omega}} \times \mathbf{r} = 0$ .

Other terms can also be calculated except,  $\mathbf{a}_{\text{rel}} = \ddot{x}\mathbf{i}$ .

Substituting all values in the first expression  $\dot{\boldsymbol{\omega}}_{CA}$  and  $\ddot{\mathbf{x}}$  and then  $\mathbf{a}_A$  can be calculated.