

# ME232: Dynamics

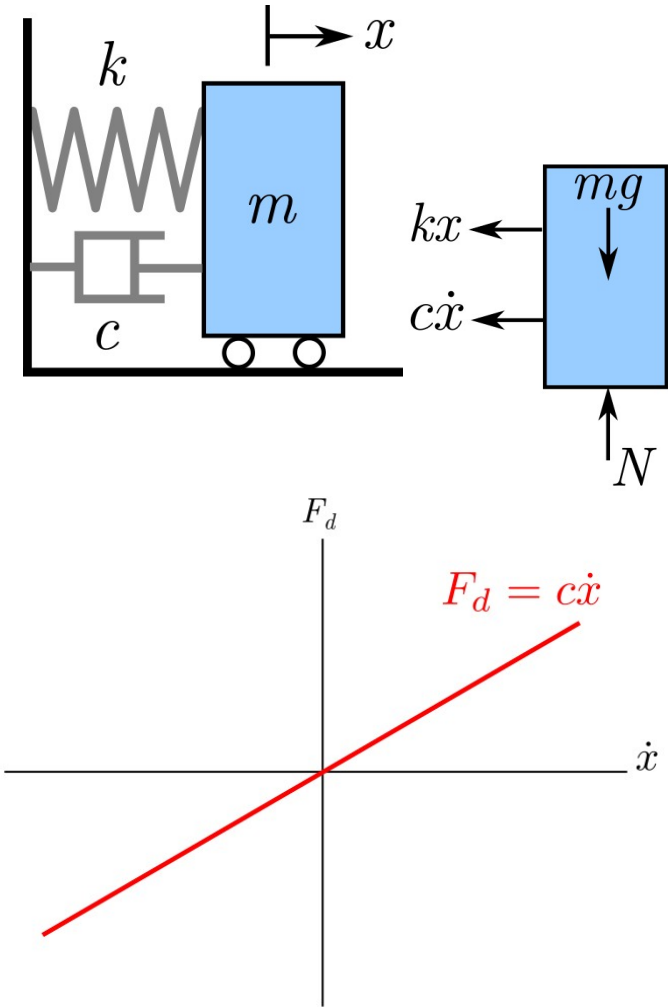
## Vibration

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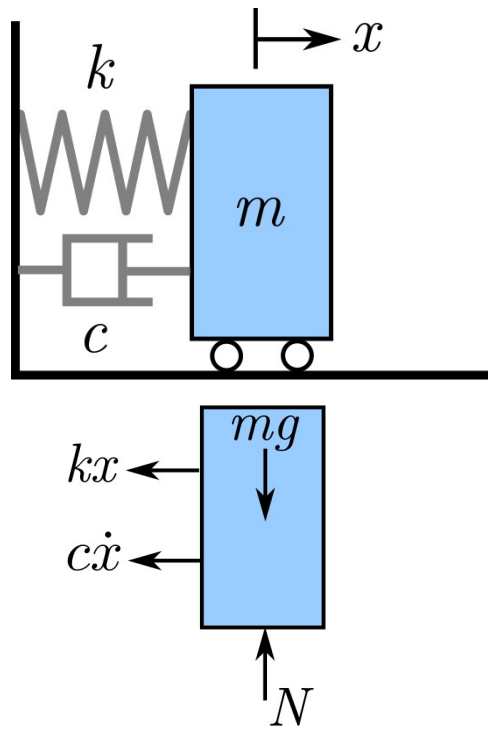
Room # 106

# Damped free vibration



Every mechanical system possesses some forces which dissipates mechanical energy. A dashpot or viscous damper is a device added to systems for limiting or retarding vibration. It consists of a cylinder filled with a viscous fluid and a piston with holes or other passages by which the fluid can flow from one side of the piston to the other.

A simple linear dashpot is shown, which exert a force  $F_d$  whose magnitude is proportional to the velocity of the mass. The constant of proportionality  $c$  is called the **viscous damping coefficient** and has units of N·s/m. The direction of the damping force applied to the mass is opposite that of the velocity  $\dot{x}$ . Thus, the force on the mass is  $-c\dot{x}$ .



The equation of motion for the body with damping is given by Newton's second as

$$-kx - c\dot{x} = m\ddot{x} \quad \text{or} \quad m\ddot{x} + c\dot{x} + kx = 0. \quad \dots\dots\dots(10)$$

In addition to the substitution  $\omega_n = \sqrt{k/m}$ , it is convenient to introduce the combination of constants  $\zeta = c/(2m\omega_n)$ . The quantity  $\zeta$  (zeta) is called the **viscous damping factor or damping ratio** and is a measure of the severity of the damping. It should be noted that  $\zeta$  is non-dimensional. (10) may now be written as

$$\ddot{x} + 2\zeta\omega_n\dot{x} + \omega_n^2 x = 0. \quad \dots\dots\dots(11)$$

In order to solve the equation of motion (11) we assume solutions of the form

$$x = Ae^{\lambda t}. \quad \dots\dots\dots(12)$$

Substituting (12) in (11) yields,  $\lambda^2 + 2\zeta\omega_n\lambda + \omega_n^2 = 0. \quad \dots\dots\dots(13)$

which is called **the characteristic equation**. Its roots are

$$\lambda_1 = \omega_n(-\zeta + \sqrt{\zeta^2 - 1}), \quad \lambda_2 = \omega_n(-\zeta - \sqrt{\zeta^2 - 1}). \quad \dots\dots\dots(14)$$

Linear systems have the **property of superposition**, which means that the general solution is the **sum of the individual solutions** each of which corresponds to one root of the characteristic equation. Thus, the general solution is

$$x = A_1 e^{\lambda_1 t} + A_2 e^{\lambda_2 t} = A_1 e^{(-\zeta + \sqrt{\zeta^2 - 1})\omega_n t} + A_2 e^{(-\zeta - \sqrt{\zeta^2 - 1})\omega_n t}. \quad \dots\dots\dots(14)$$

*Categories of damped motion:*

Because  $0 \leq \zeta \leq \infty$ , the radicand  $(\zeta^2 - 1)$  may be positive, negative, or even zero, giving rise to the following three categories of damped motion:

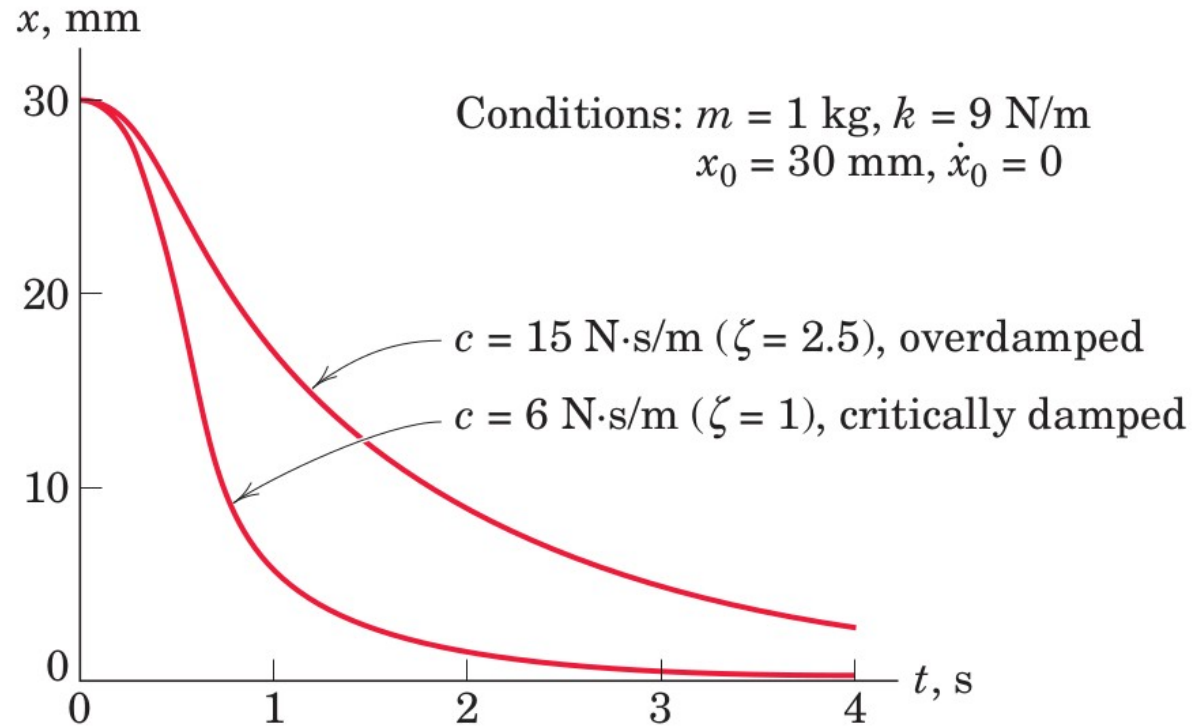
**I.  $\zeta > 1$  (overdamped):** The roots  $\lambda_1$  and  $\lambda_2$  are **distinct, real, and negative numbers**. The motion as given by (14) decays so that  **$x$  approaches zero for large values of time  $t$** . There is **no oscillation** and therefore no period associated with the motion.

**II.  $\zeta = 1$  (critically damped):** The roots  $\lambda_1$  and  $\lambda_2$  are **equal, real, and negative numbers** ( $\lambda_1 = \lambda_2 = -\omega_n$ ). The solution to the differential equation for the special case of equal roots is given by

$$x = (A_1 + A_2 t)e^{-\omega_n t}. \quad \dots\dots\dots(15)$$

Again, the motion **decays with  $x$  approaching zero for large  $t$** , and the motion is nonperiodic.

A critically damped system, when excited with an initial velocity or displacement (or both), will approach equilibrium faster than will an overdamped system.



III.  $\zeta < 1$  (**underdamped**): So the radicand  $(\zeta^2 - 1)$  is negative and we may rewrite (14) as

$$x = e^{-\zeta\omega_n t}[A_1e^{i\sqrt{1-\zeta^2}\omega_n t} + A_2e^{-i\sqrt{1-\zeta^2}\omega_n t}].$$

It is convenient to let a new variable  $\omega_d$  represent the combination  $\omega_n\sqrt{1-\zeta^2}$  . Thus,

$$x = e^{-\zeta\omega_n t}[A_1e^{i\omega_d t} + A_2e^{-i\omega_d t}]. \hspace{10em} \text{.....(16a)}$$

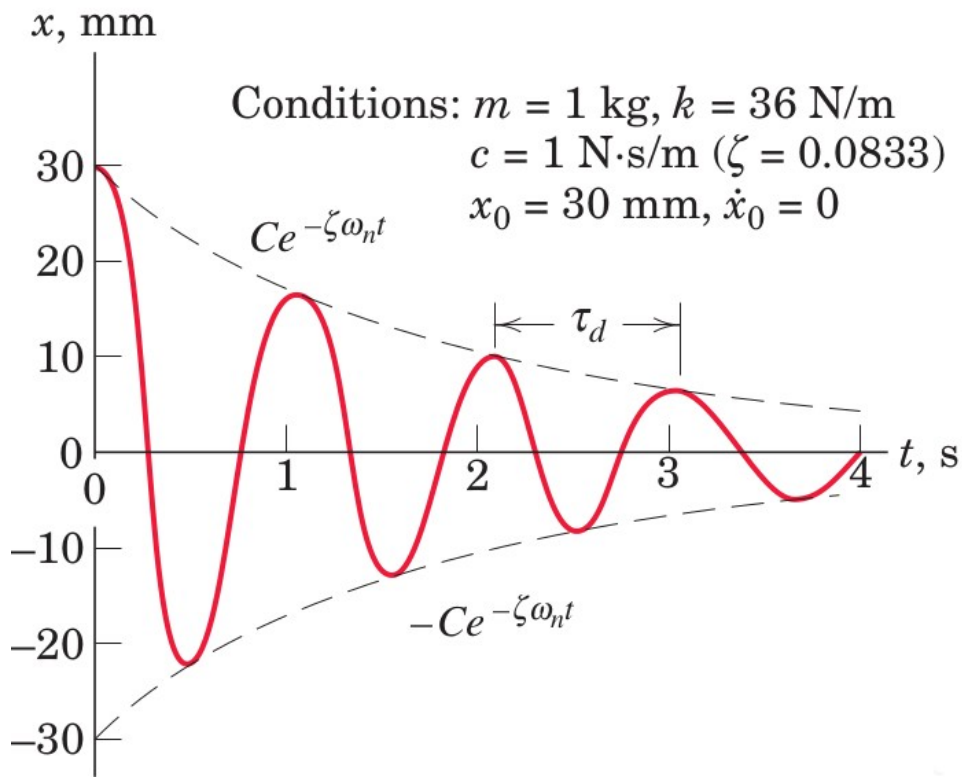
Use of the Euler formula (16) can be rewritten as

$$x = e^{-\zeta\omega_n t}[A_3 \cos \omega_d t + A_4 \sin \omega_d t], \hspace{10em} \text{.....(16b)}$$

where  $A_3 = (A_1 + A_2)$  and  $A_4 = i(A_1 - A_2)$ . Alternatively we can also write,

$$x = e^{-\zeta\omega_n t}[C \sin(\omega_d t + \psi)], \hspace{10em} \text{.....(16c)}$$

or 
$$x = Ce^{-\zeta\omega_n t} \sin(\omega_d t + \psi).$$



(16) represents an exponentially decreasing harmonic function, as shown in figure for specific numerical values. The frequency

$$\omega_d = \omega_n \sqrt{1 - \zeta^2}$$

is called the **damped natural frequency**. The damped period is given by  $\tau_d = 2\pi/\omega_d$ .

To find  $C$  and  $\psi$  if damping is present we use (16) and apply initial conditions, i.e., at  $t = 0$ , initial displacement is  $x_0$  and initial velocity is  $\dot{x}_0$ , respectively.