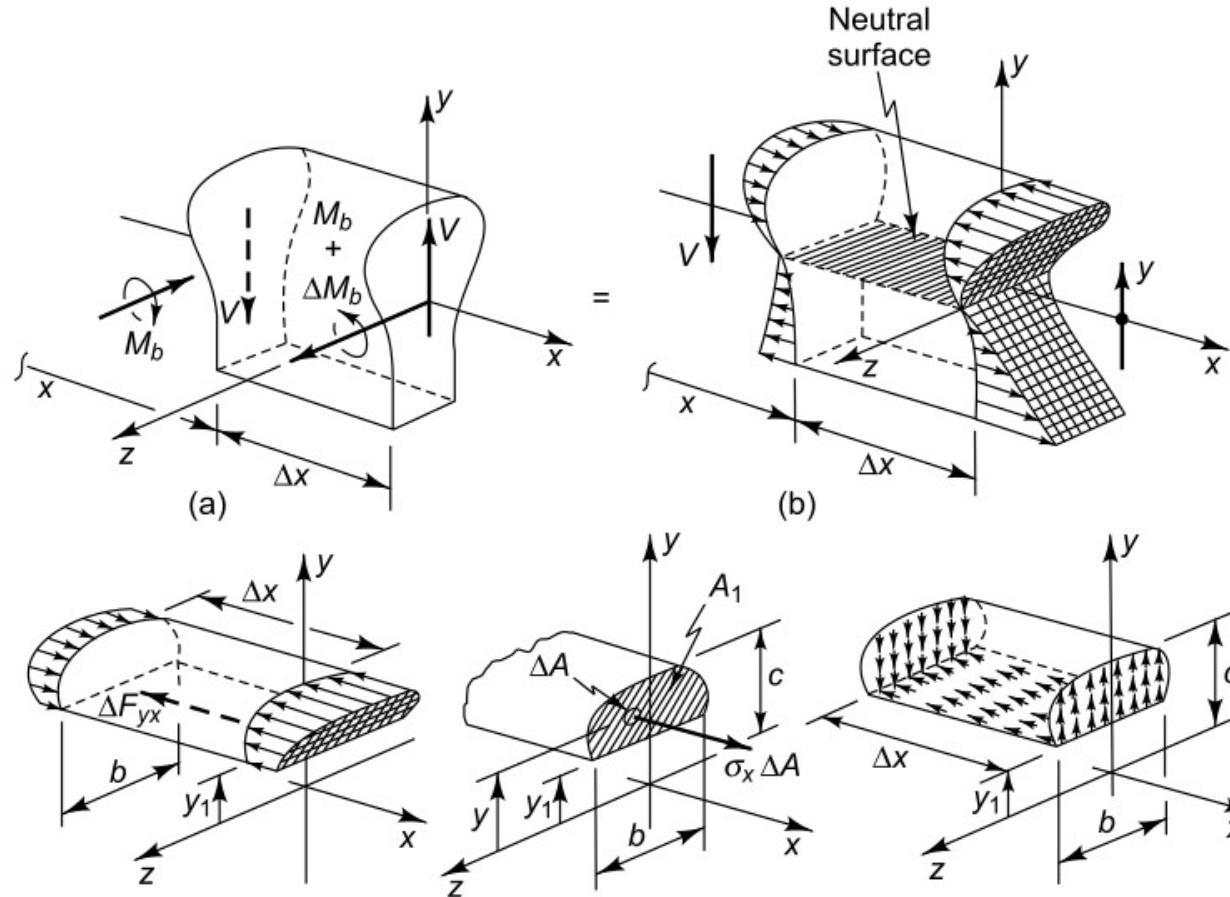


# **ME231: Solid Mechanics-I**

## **Stresses due to bending**

# Stresses in symmetrical elastic beams transmitting both Shear force and bending moment



Shear stress at  $y=y_1$  is,

$$\tau_{yx} = \frac{VQ}{bI_{zz}} = \tau_{xy},$$

where,

$$Q = \int_{A_1} y dA.$$

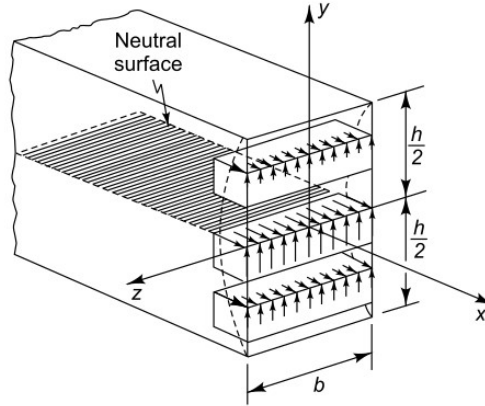
Shear flow due to bending,

$$q_{yx} = b\tau_{yx} = \frac{VQ}{I_{zz}}.$$

# Shear stress distribution in rectangular beams

$$\frac{\partial \sigma_x}{\partial x} + \frac{\partial \tau_{xy}}{\partial y} = 0$$

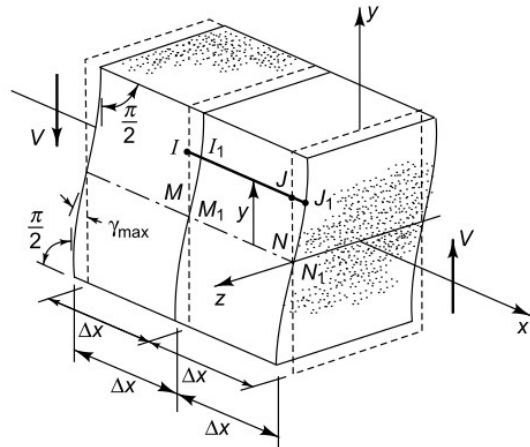
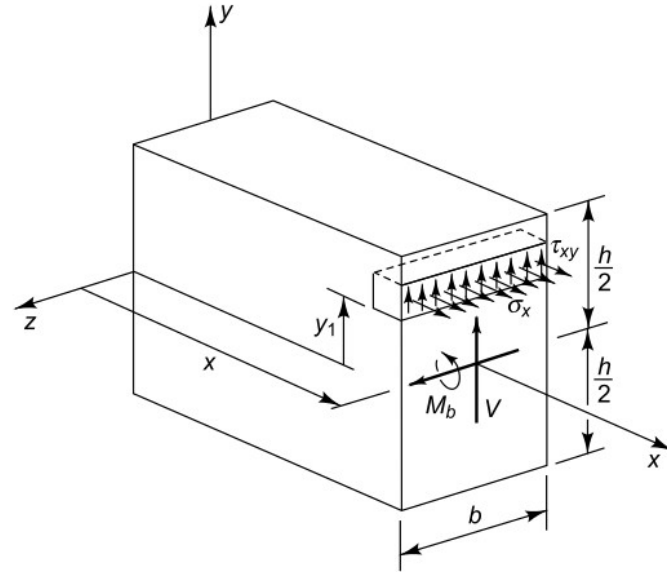
$$\frac{\partial \tau_{xy}}{\partial x} + \frac{\partial \sigma_y}{\partial y} = 0$$



$$\tau_{xy} = \frac{V}{2I_{zz}} \left[ \left( \frac{h}{2} \right)^2 - y_1^2 \right]$$

or

$$\tau_{xy} = \frac{3V}{2A} \left[ 1 - \left( \frac{y_1}{h/2} \right)^2 \right]$$



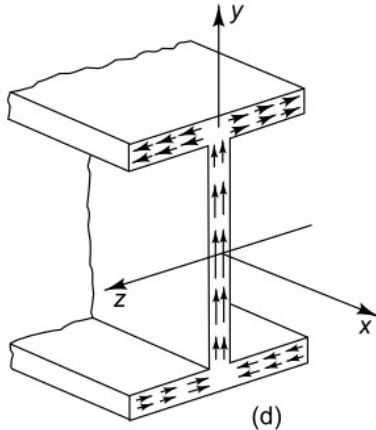
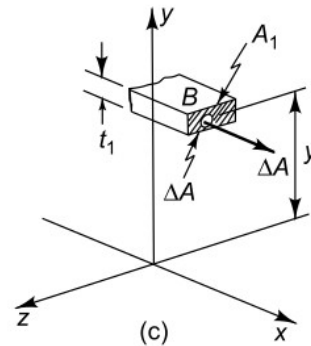
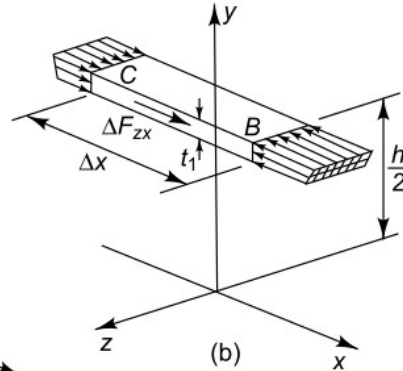
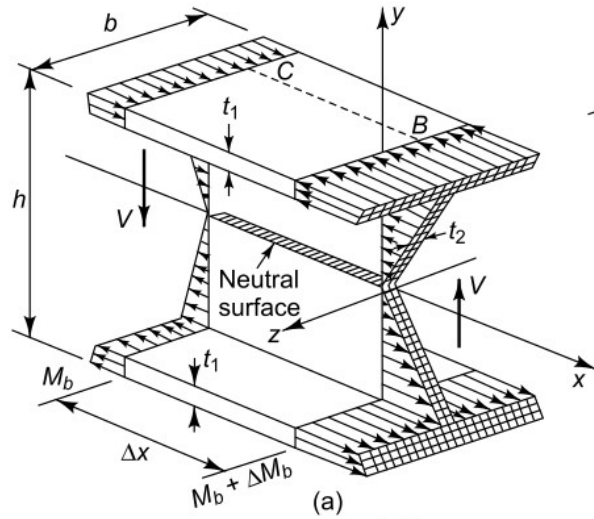
$$(\tau_{xy})_{\text{avg}} = \frac{3V}{2A}$$

(If exact stress variation is assumed)

$$(\tau_{xy})_{\text{avg}} = \frac{V}{A}$$

(If constant stress variation is assumed)

# Shear stress distribution in I-beams



Shear flow,

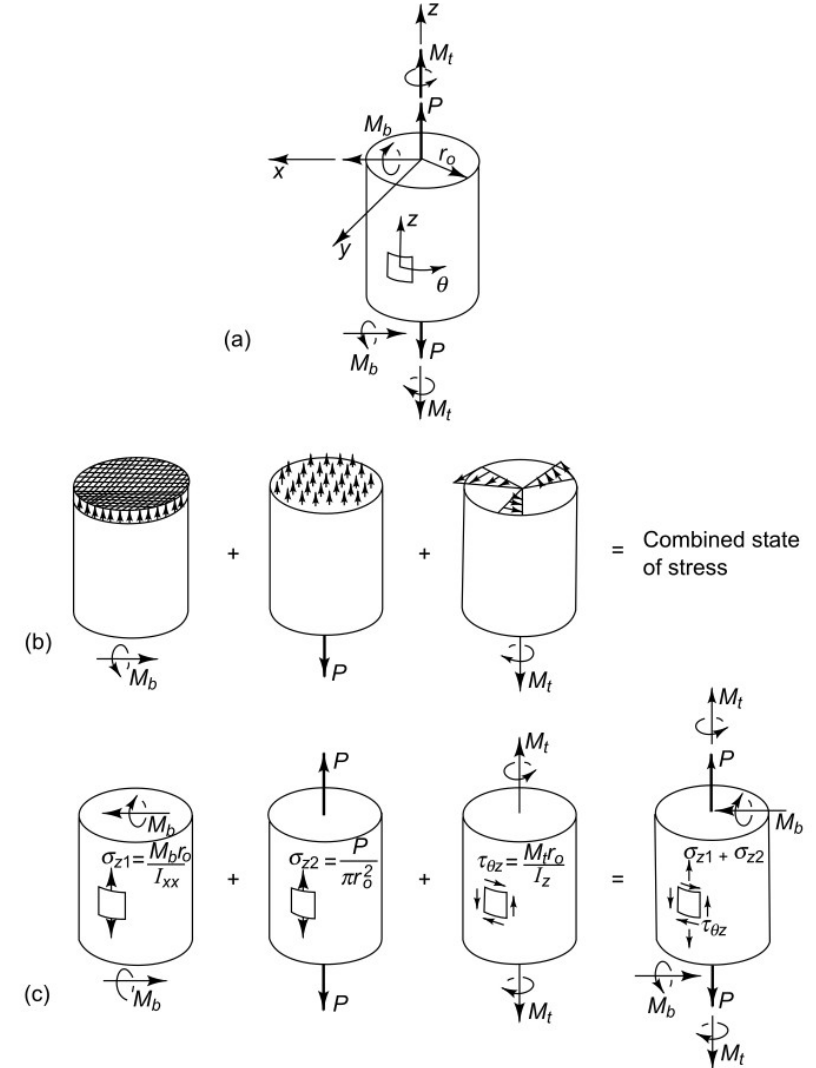
$$q_{zx} = t_1 \tau_{zx} = -\frac{VQ}{I_{zz}}.$$

Shear stress at point B,

$$\tau_{zx} = -\frac{VQ}{t_1 I_{zz}} = \tau_{xz}.$$

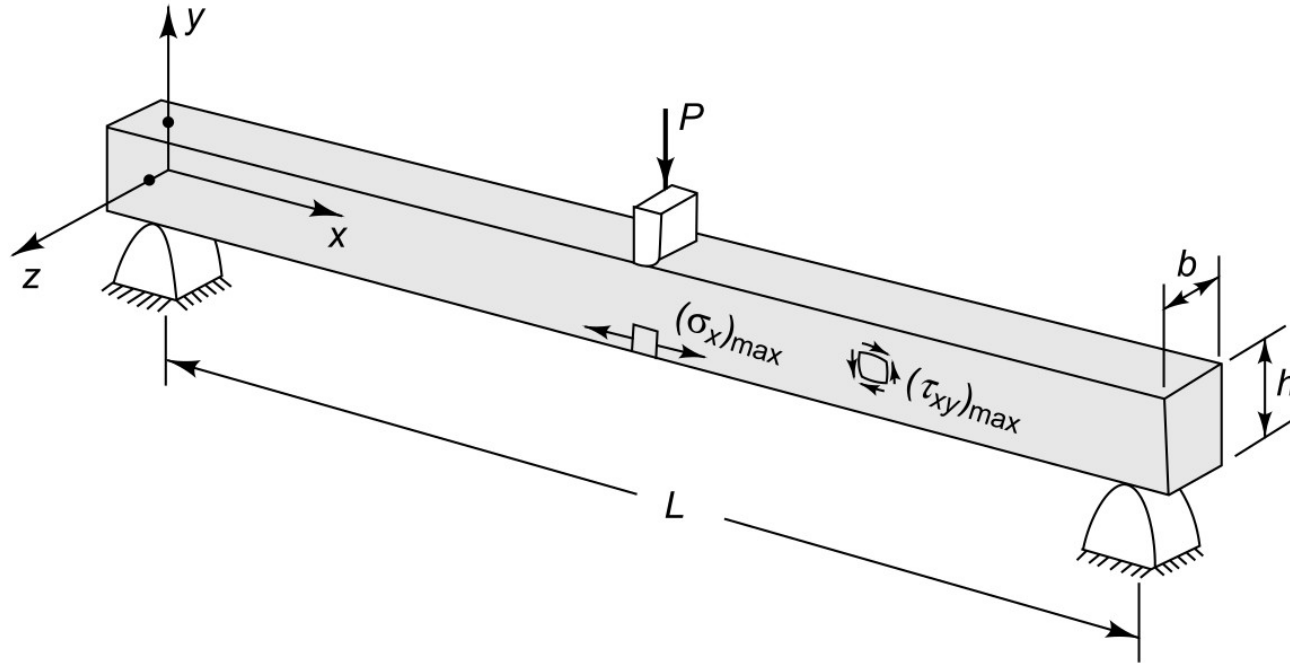
# Example 3: Stress analysis in bending-Combined stresses

In figure an elastic circular shaft is shown transmitting simultaneously a bending moment  $M_b$ , an axial tensile force  $P$ , and a twisting moment  $M_t$ . We wish to study the state of combined stress.



## Example 4

A rectangular beam is carried on simple supports and subjected to a central load. We wish to find the ratio of the maximum shear stress  $(\tau_{xy})_{\max}$  to the maximum bending stress  $(\sigma_x)_{\max}$ .



# Strain energy in bending

## Pure bending

$$U = \frac{1}{2} \iiint \sigma_x \epsilon_x dx dy dz = \iiint \frac{\sigma_x^2}{2E} dx dy dz$$

$$U = \iiint \frac{1}{2E} \left( \frac{M_b y}{I_{zz}} \right)^2 dx dy dz = \int_L \frac{M_b^2}{2EI_{zz}} dx \int_A y^2 dy dz$$

$$U = \int_L \frac{M_b^2}{2EI_{zz}} dx$$

## Bending with transverse loads

$$\begin{aligned} U &= \frac{1}{2} \iiint (\sigma_x \epsilon_x + \tau_{xy} \gamma_{xy} + \tau_{xz} \gamma_{xz}) dx dy dz \\ &= \iiint \frac{\sigma_x^2}{2E} dx dy dz + \iiint \frac{\tau_{xy}^2 + \tau_{xz}^2}{2G} dx dy dz \end{aligned}$$