#### ME531: Advanced Mechanics of Solids

# Motion, Strain and Stress

 $Anshul\ Faye$  afaye@iitbhilai.ac.in  $Room\ No.\ \#\ 106$ 

#### Measure of Deformation is solids

From our knowledge of Strength of materials (UG course), we know that in linear stress-strain analysis the deformation of a continuum body is measured in terms of small strains.

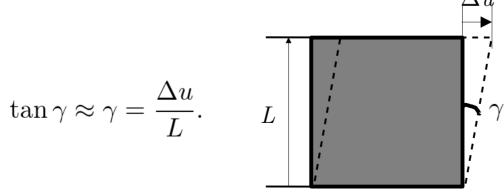
For instance, small strain in axial loading is defined as

$$\epsilon = \frac{\Delta l}{L},$$

$$L \qquad \Delta l$$

where L is the initial length and  $\Delta l$  is the elongation.

Similarly, shear strain is defined as



### Small strains in 2D

$$\varepsilon_{11} = \lim_{\Delta x_1 \to 0} \frac{A'B' - AB}{AB} = \lim_{\Delta x_1 \to 0} \frac{\Delta x_1 + \frac{\partial u}{\partial x_1} \Delta x_1 - \Delta x_1}{\Delta x_1},$$

$$\Rightarrow \varepsilon_{11} = \lim_{\Delta x_1 \to 0} \frac{A'B' - AB}{AB} = \lim_{\Delta x_1 \to 0} \frac{\Delta x_1 + \frac{\partial u}{\partial x_1} \Delta x_1 - \Delta x_1}{\Delta x_1},$$

$$\Rightarrow \varepsilon_{11} = \frac{\partial u_1}{\partial x_1},$$

$$\varepsilon_{22} = \lim_{\Delta x_2 \to 0} \frac{A'D' - AD}{AD} = \lim_{\Delta x_2 \to 0} \frac{\Delta x_2 + \frac{\partial u_2}{\partial x_2} \Delta x_2 - \Delta x_2}{\Delta x_2},$$

$$\Rightarrow \varepsilon_{22} = \frac{\partial u_2}{\partial x_2},$$

$$\psi_{12} = \lim_{\Delta x_1 \to 0} \frac{\pi}{2} - \angle B'A'D' = \alpha + \beta$$

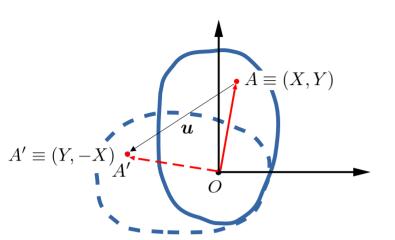
$$\Rightarrow \gamma_{12} = \lim_{\Delta x_1 \to 0} \frac{\partial u_2}{\partial x_1} \Delta x_1 + \frac{\partial u_1}{\partial x_2} \Delta x_2$$

$$\Rightarrow \gamma_{12} \approx \frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \Delta x_1 + \frac{\partial u_2}{\partial x_2} \Delta x_2$$

$$\Rightarrow \gamma_{12} \approx \frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \Delta x_1$$

$$\Rightarrow \gamma_{12} \approx \frac{\partial u_1}{\partial x_2} + \frac{\partial u_2}{\partial x_1} \Delta x_1 + \frac{\partial u_2}{\partial x_2} \Delta x_2$$

### What if displacements are large?



Consider a body which goes under a rotation of  $90^{\circ}$  in the anti-clockwise direction. During the rotation, the point A which had its coordinates as (X,Y) before the rotataion, take a new position A' with its coordinates as (-Y,X) after the rotation.

Displacements of point A along X and Y directions are,

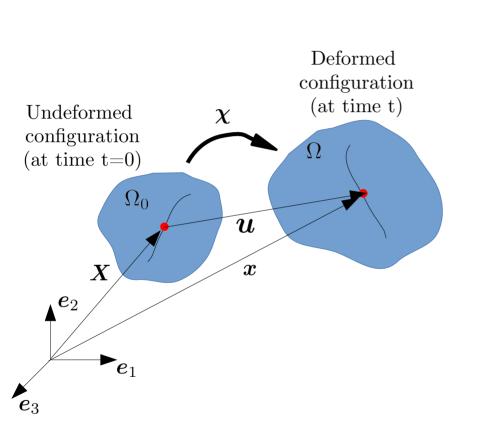
$$u_x = Y - X$$
 and  $u_y = -X - Y$ .

Let us determine the components of strain for this case as,

$$\varepsilon_x = \frac{\partial u_x}{\partial X} = -1, \quad \varepsilon_Y = \frac{\partial u_y}{\partial Y} = -1, \quad \text{and} \quad \gamma_{xy} = \frac{\partial u_x}{\partial Y} + \frac{\partial u_y}{\partial X} = 0.$$

Are this strains correct?

#### Motion and Deformation



Consider a body in its undeformd (or initial) configuration with a volume  $\Omega_0$ . After time t, the body moves to a new position occupying a volume  $\Omega$  in the space. The configuration at time t is called deformed (or current) configuration.

A function  $\chi$ , which maps a point X in  $\Omega_0$  to another point x in  $\Omega$  is called *motion* and the motion is assumed to be uniquely *invertible*, i.e.

$$\boldsymbol{x} = \boldsymbol{\chi}(\boldsymbol{X}, t) \text{ and } \boldsymbol{X} = \boldsymbol{\chi}^{-1}(\boldsymbol{x}, t).$$

It can be observed that, x(X,t) = X + u(X,t), where u is the displacement vector.

### Lagrangian and Eulerian Description

Charecterization of motion (or any other quantity) with respect to material (or referential) coordinates  $X_i$  is called *material* or *Lagrangian* descrition. *Spatial* or *Eulerian* description is the characterized with respect to spatial (or current) coordinates  $x_i$ .

E.g. Displacement field in the Lagrantial form is  $\boldsymbol{U}(\boldsymbol{X},t) = \boldsymbol{x}(\boldsymbol{X},t) - \boldsymbol{X}$ , and Eulerian form of the displacement is  $\boldsymbol{u}(\boldsymbol{x},t) = \boldsymbol{x} - \boldsymbol{X}(\boldsymbol{x},t)$ .

For a very simple motion given as, 
$$x_1(X_1, X_2, X_3, t) = X_1 + \gamma X_2,$$
  $x_2(X_1, X_2, X_3, t) = X_2,$   $x_3(X_1, X_2, X_3, t) = X_3,$ 

Material description of displacement is

$$U_1(X_1, X_2, X_3, t) = x_1 - X_1 = \gamma X_2,$$
  

$$U_2(X_1, X_2, X_3, t) = 0,$$
  

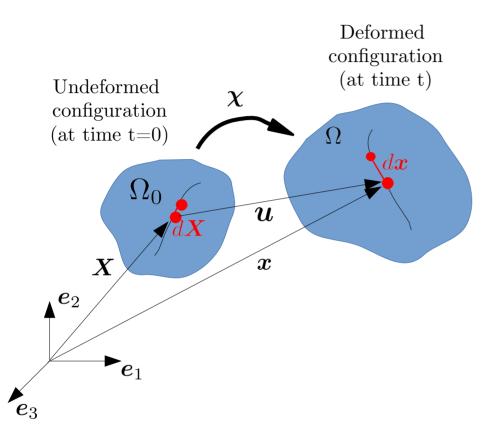
$$U_3(X_1, X_2, X_3, t) = 0,$$

and special description of is

$$u_1(x_1, x_2, x_3, t) = x_1 - X_1 = \gamma X_2 = \gamma x_2,$$
  
 $u_2(x_1, x_2, x_3, t) = 0,$   
 $u_3(x_1, x_2, x_3, t) = 0.$ 

It should be noticed that u(x,t) = U(X,t).

# Deformation gradient



Consider a line element  $d\mathbf{X}$  in  $\Omega_0$  which deforms to a line element  $d\mathbf{x}$ . Deformation gradient tensor maps the line element  $d\mathbf{X}$  to  $d\mathbf{x}$  as,

$$d\mathbf{x} = \mathbf{F}d\mathbf{X} \text{ or } dx_i = F_{iJ}dX_J,$$
  
where,  $\mathbf{F} = \frac{\partial \mathbf{x}}{\partial \mathbf{X}} \text{ or } F_{iJ} = \frac{\partial x_i}{\partial X_J},$   
or  $\mathbf{F} = \text{Grad } \mathbf{x} = \nabla_X \mathbf{x}$ 

is an invertible tensor.  $F^{-1}$  is the *inverse* deformation gradient tensor, defined as

$$F^{-1} = \frac{\partial \mathbf{X}}{\partial \mathbf{x}} \text{ or } F_{Ij}^{-1} = \frac{\partial X_I}{\partial x_j}.$$
  
or  $F^{-1} = \text{grad } \mathbf{X} = \nabla \mathbf{X}$ 

It relates line elements as,

$$d\mathbf{X} = \mathbf{F}^{-1}d\mathbf{x}$$
 or  $dX_I = F_{Ii}^{-1}dx_j$ .

### Example

Consider a two dimensional motion given by two equations as

$$x_1 = 4 - 2X_1 - X_2$$
$$x_2 = 2 + 1.5X_1 - 0.5X_2$$

Deformation gradient for the given motion is calculated as,

$$[\mathbf{F}] = \begin{bmatrix} -2 & -1 \\ 1.5 & -0.5 \end{bmatrix}, \text{ and } [\mathbf{F}]^{-1} = \frac{1}{5} \begin{bmatrix} -1 & 2 \\ -3 & -4 \end{bmatrix}$$

Now consider following vectors  $\boldsymbol{a}_0 = [1,0], \ \boldsymbol{b}_0 = [0,1]$  and  $\boldsymbol{c}_0 = [0.707,\ 0.707]$  in the reference configuration. Current or deformed vectors can be obtained as  $\boldsymbol{a} = \boldsymbol{F}\boldsymbol{a}_0$ ,  $\boldsymbol{b} = \boldsymbol{F}\boldsymbol{b}_0$  and  $\boldsymbol{c} = \boldsymbol{F}\boldsymbol{c}_0$ .

Consider another vector  $\mathbf{d}=[1,0]$  in the current configuration. Reference configuration  $\mathbf{d}_0$  for vector d can be obtained as  $\mathbf{d}_0 = \mathbf{F}^1 \mathbf{d}$ .

