

Ex1 Consider the mass-spring-damper system in the notes 5.1, 5.2, p 82, with force  $\{F_t: t \geq 0\}$  being white noise with a given intensity  $S_{FF}(\omega) = \sigma^2$ .

Write the system in the standard form  $dx_t = Ax_t dt + G dB_t$ .

$$A = \begin{bmatrix} 0 & 1 \\ -k/m & -c/m \end{bmatrix} \quad G = \sigma \begin{bmatrix} 0 \\ 1/m \end{bmatrix} = \begin{bmatrix} 0 \\ \sigma/m \end{bmatrix}$$

Ex2 Using the general form, simulate the system on the time interval  $t \in [0, 1000]$  using the Euler method. Take system parameters  $m = 1 \text{ kg}$ ,  $k = \frac{1}{2} \text{ N/m}$ ,  $c = 0.2 \text{ Ns/m}$  and  $\sigma^2 = 100 \text{ N}^2 \text{ s}$ . Let the system start at rest at  $t = 0$ . Use a time step of  $\Delta t = 0.01 \text{ s}$ .

See R.

Ex3 Estimate from your simulation the steady-state variance of position  $Q_t$ , of velocity  $V_t$ , and the covariance between the two. Compare with the solution of the algebraic Lyapunov equation governing the variance.

• From the Sylvester equation (see wiki) we know

$$AX + XB = C \Leftrightarrow (I_m \otimes A + B^T \otimes I_n) \text{vec } X = \text{vec } C$$

Where  $\text{vec}(\cdot)$  is the vectorization operator. Hence the Lyapunov equation, eq 5.27, can be rewritten as follows

$$A\Sigma + \Sigma A^T + G G^T = 0 \Rightarrow$$

$$A\Sigma + \Sigma A^T = -G G^T \Rightarrow$$

$$(I_n \otimes A + A \otimes I_n) \text{vec } \Sigma = -\text{vec}(G G^T) \Rightarrow$$

$$\text{vec } \Sigma = -(I_n \otimes A + A \otimes I_n)^{-1} \text{vec}(G G^T)$$

Hence we can solve for the analytical covariance matrix  $\Sigma$

$$\Sigma = \begin{bmatrix} 500 & 0 \\ 0 & 250 \end{bmatrix}$$

See the rest in R

Ex4 The kinetic energy is  $\frac{1}{2}mV_t^2$  while the potential energy is  $\frac{1}{2}kQ^2$ . In steady-state, what is the expected kinetic and potential energy?

• We first calculate the potential energy

$$\begin{aligned} E[\frac{1}{2}kQ_t^2] &= \frac{1}{2} \cdot \frac{1}{2} E[Q_t^2] & \left( \begin{aligned} V[Q] &= E[Q^2] - E[Q]^2 \\ E[Q^2] &= V[Q] + E[Q]^2 \end{aligned} \right) \\ &= \frac{1}{4}(V[Q] + E[Q]^2) & (E[Q] = 0 \text{ in steady state}) \\ &= \frac{1}{4}V[Q] \end{aligned}$$

from the previous exercise we know  $V[Q] = 500$  so

$$E[\frac{1}{2}kQ_t^2] = 125$$

Next the kinetic energy

$$\begin{aligned} E[\frac{1}{2}mV_t^2] &= \frac{1}{2} \cdot 1 \cdot E[V_t^2] \\ &= \frac{1}{2}V[V_t] \end{aligned}$$

From the previous exercise we know  $V[V_t] = 250$  so

$$E[\frac{1}{2}mV_t^2] = 125$$

This makes sense because the kinetic and potential energy should be equal in the steady state.



Ex 5 For the simulation, compute and plot the empirical acf of  $\{Q_t\}$  up to lag 50. Add to the plot the theoretical acf.

• We find the theoretical acf on page 111 as eq. 5.28.

$$\rho(t) = \sum \exp(A^T t), \text{ for } t \geq 0$$

Hence in our case

$$\rho(t) = \begin{bmatrix} 500 & 0 \\ 0 & 250 \end{bmatrix} \exp\left(\begin{bmatrix} 0 & -1/2 \\ 1 & -1/5 \end{bmatrix} t\right)$$

The matrix exponential is possible to calculate analytically but it is very nasty. Hence we will not write it here but only calculate it numerically in R using the function and package expm. See the rest in R.

Ex 6 Plot as a function of the frequency  $\omega$ , the amplitude and phase of the frequency response from the noise to the position. Plot also the theoretical variance spectrum of the position.

- On page 96 the noise is substituted with a force but it could just as well have been a noise term. The frequency response from noise to position (in the book from external force to internal state,  $U_b \rightarrow X_b$ ), is given as

$$H(\omega) = (i\omega \cdot I - A)^{-1} G,$$

for linear systems.

Hence for our system

$$H(\omega) = \frac{100}{2i\omega - 10\omega^2 + 5} \begin{bmatrix} 1 \\ i\omega \end{bmatrix}$$

where the position is the first entry and the velocity is the second.

On page 104, eq 5.14 the variance spectrum is given as

$$S_x(\omega) = H(-\omega) \cdot S_u(\omega) \cdot H^T(\omega)$$

where  $S_u$  is the white noise spectrum. It is in this exercise given as

$$S_u(\omega) = S_{FF}(\omega) = \sigma^2, \text{ so}$$

$$S_x(\omega) = H(-\omega) \cdot \sigma^2 \cdot H^T(\omega)$$

$$= H(\omega) \cdot 100 \cdot H^T(\omega)$$

$$= \begin{bmatrix} \frac{-10^6}{-100\omega^4 + 96\omega^2 - 25} & \frac{-10^6 i\omega}{-100\omega^4 + 96\omega^2 - 25} \\ \frac{10^6}{-100\omega^4 + 96\omega^2 - 25} & \frac{-10^6 \omega^2}{-100\omega^4 + 96\omega^2 - 25} \end{bmatrix}$$

where the variance spectrum for the position is given is  $S_x(\omega)[1,1]$  i.e.

$$S_q(\omega) = S_x(\omega)[1,1] = \frac{-10^6}{-100\omega^4 + 96\omega^2 - 25}$$

See plots in R.



Variance in a scalar linear system  
Consider the scalar linear system

$$\dot{X}_t = aX_t + gU_t, \quad X_0 = x$$

where  $\{U_t : t \geq 0\}$  is Gaussian white noise, i.e. the formal derivative of standard Brownian motion.

Ex 7. Write up the mean  $E[X_t]$  as a function of time.

• We know that

$$\begin{aligned}\dot{X} &= AX + GU \\ \Downarrow \\ X_t - X_0 &= \int_0^t \dot{X}_s ds = \int_0^t AX_s + GU_s ds \\ &= \int_0^t AX_s ds + \int_0^t GU_s ds \\ &= \int_0^t AX_s ds + GB_t \quad \left[ \text{because } U \text{ is white noise and it} \right. \\ &\quad \left. \text{is the derivative of brownian motion} \right]\end{aligned}$$

Hence we also know that

$$\begin{aligned}\frac{d}{dt} E[X_t] &= A E[X_t] + G E[B_t] \\ &= A E[X_t]\end{aligned}$$

Hence we know that  $E[X]$  is just the deterministic solution to the differential equation.

$$E[X_t] = x e^{at}$$

Ex8 Write up the differential Lyapunov equation governing the variance  $\text{Var}[X_e]$ , and solve it.

- The differential Lyapunov equation is given as eq (5.21) in the book and rephrased here in scalar form.

$$\begin{aligned}\frac{d\Sigma(t)}{dt} &= a\Sigma(t) + \Sigma(t)a + g^2 \\ &= 2a\Sigma(t) + g^2\end{aligned}$$

We solve the differential equation with  $\Sigma(0) = C$

$$\Sigma(t) = e^{2at} \left( C + \frac{g^2}{2a} \right) - \frac{g^2}{2a}$$

using the formula

$$\frac{dx}{dt} = ax + b, \quad x(0) = C \Rightarrow x(t) = e^{at} \left( C + \frac{b}{a} \right) - \frac{b}{a}$$

Ex9. Assume that the system is stable. What is the steady-state variance,  $\lim_{t \rightarrow \infty} \text{Var}[x_t]$

- The stationary variance is given by equation 5.27 and here given for the scalar case.

$$2a\Sigma + g^2 = 0 \Rightarrow \Sigma = -\frac{g^2}{2a}$$

Ex10 Verify that the steady-state variance is a equilibrium point of the Lyapunov equation.

- An equilibrium point is a point where no change will happen i.e.

$$\frac{d\Sigma}{dt} = 0$$

We check

$$\begin{aligned} \frac{d\Sigma}{dt} &= 2a\Sigma + g^2 \\ &= 2a(-g^2/2a) + g^2 \\ &= 0 \end{aligned}$$