

1 Problem Description

2 $G_p(s)$ Analysis

3 Feedback Design

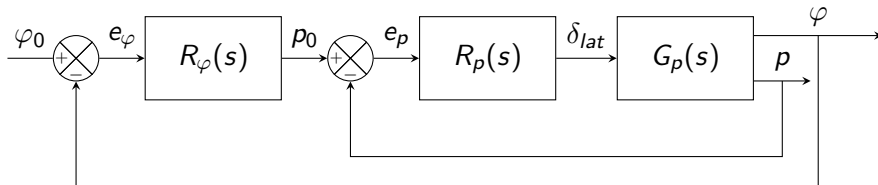
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Lateral Attitude Control Plant



Blocks:

- **Roll angle:** $R_\varphi(s) = K_\varphi$; **P** controller
- **Roll rate:** $R_p(s) = K_{p,P} + \frac{K_{p,I}}{s} + \frac{K_{p,D}}{1+sT_p} s$; **PID** controller
- **Lateral dynamics system:** $G_p(s)$;

Lateral Dynamics System

The lateral dynamics system is described by:

$$\begin{bmatrix} \dot{\mathbf{x}} \\ \mathbf{y} \end{bmatrix} = \begin{bmatrix} \mathbf{A} \\ \mathbf{C} \end{bmatrix} \mathbf{x} + \begin{bmatrix} \mathbf{B} \\ \mathbf{D} \end{bmatrix} u$$

where:

$$\mathbf{y} = \begin{bmatrix} p \\ \varphi \end{bmatrix}, \mathbf{x} = \begin{bmatrix} v \\ p \\ \varphi \end{bmatrix}, u = \delta_{lat}$$

and:

$$\mathbf{A} = \begin{bmatrix} Y_v & Y_p & g \\ L_v & L_p & 0 \\ 0 & 1 & 0 \end{bmatrix}, \mathbf{B} = \begin{bmatrix} Y_\delta \\ L_\delta \\ 0 \end{bmatrix}, \mathbf{C} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}, \mathbf{D} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$



Parameters & Uncertainties

All the uncertainties are given in terms of **standard deviation**, σ , referred to a **Gaussian** distribution.

Stability derivatives

- $Y_v = -0.264 \frac{1}{s} (4.837\%)$
- $Y_p = 0 \frac{m}{s \text{ rad}} (0\%)$
- $L_v = -7.349 \frac{\text{rad}}{m} s (4.927\%)$
- $L_p = 0 \frac{1}{s} (0\%)$
- $g = 9.806 \frac{m}{s^2} (0\%)$

Control derivatives

- $Y_\delta = 9.568 \frac{m}{s^2} (4.647\%)$
- $L_\delta = 1079.339 \frac{\text{rad}}{s^2} (2.672\%)$



Solution Target

- **Nominal** and **uncertain** system analysis
- **Feedback** system assembly
- **Nominal** system tuning: K_φ , $K_{p,P}$, $K_{p,I}$ & $K_{p,D}$
- **Robust** analysis and design
- **Verification** of results
 - **Uncertain model** for the feedback system
 - **Monte-Carlo** simulation
 - **Uncertain model** using real Δ and μ -analysis verification

Design requirements

- φ response to φ_0 is a 2nd order response ($\omega_n \geq 10 \frac{\text{rad}}{\text{s}}$ and $\xi \geq 0.9$)
- $|\delta_{lat}| \leq 5$ for a doublet change in φ_0 , with a 10° amplitude
- Robust stability for a $\pm 3\sigma$ uncertainty treated as deterministic bounds



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Lateral Dynamics Assembly

Poles & Zeros

Response Analysis

3 Feedback Design

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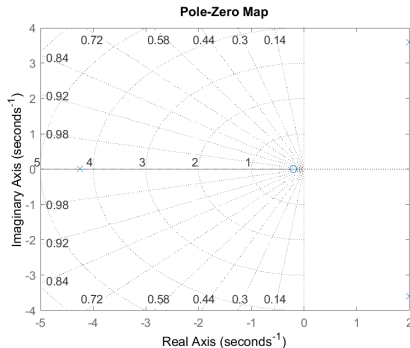
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lateralDynamics.m

```
% loading quadricopter data
load('ANTRdata.mat');
% stability derivatives -- uncertainty expressed as 3 * sigma with Gauss distribution
Y_v = ureal('Y_v', Y_v, 'percentage', 4.837*3); % 1/s      % uncertainty 4.837%
L_v = ureal('L_v', L_v, 'percentage', 4.927*3); % rad s/m  % uncertainty 4.927%
% control derivatives -- uncertainty expressed as 3 * sigma with Gauss distribution
Y_d = ureal('Y_d', Y_d, 'percentage', 4.647*3); % m/s2     % uncertainty 4.647%
L_d = ureal('L_d', L_d, 'percentage', 2.762*3); % rad/s2    % uncertainty 2.762%
%% lateral dynamics system assembly
A = [Y_v, Y_p, g; L_v, L_p, 0; 0, 1, 0];
B = [Y_d; L_d; 0];
C = [0, 1, 0; 0, 0, 1];
D = [0; 0];
%% conversion from time domain to state space domain
% lateral dynamics state space model assembly
G = ss(A, B, C, D);
% input and output declaration
% input name
G.u = '\delta_{lat}';
% output name
G.y = {'p', '\phi'};
% setting up nominal G -> nominal plant model
G_nom = G.nominal;
```

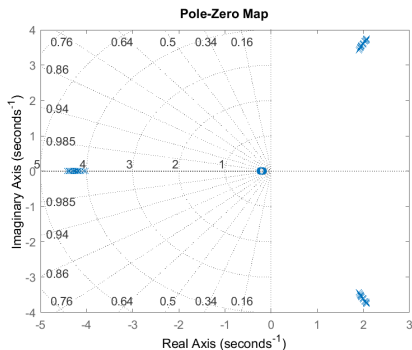
Poles & Zeros – Nominal System



- The lateral dynamics system is **unstable**, there are **2 poles** with $\Re > 0$. In order to **stabilize** the system there is the **need** of controllers.

```
% lateral dynamics matrix assembly
run lateralDynamics;
% ...
% poles/zeros study for
% the nominal configuration
pzplot(G.Nominal);
```

Poles & Zeros – Uncertain System

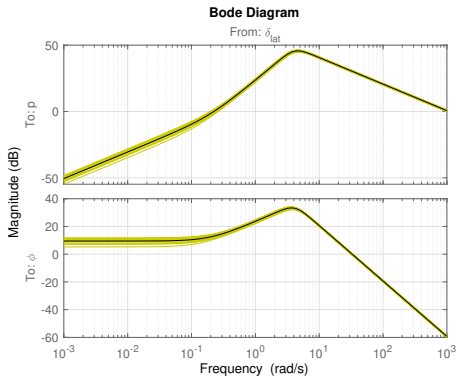


- The $\Re > 0$ poles **still** remain in the **right-hand-side** plane.
- The $\Re < 0$ pole and zero **stay** in the **left-hand-side** plane.

```
% lateral dynamics matrix assembly
run lateralDynamics;
% ...
% poles/zeros study for
% the uncertain configuration
pzplot(G);
```



Frequency Response Function



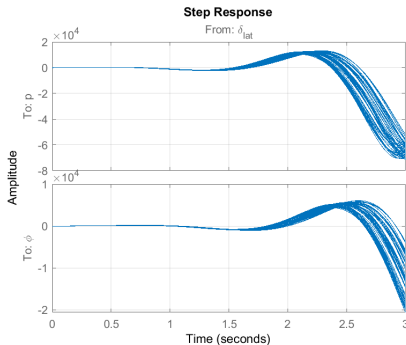
Taking into account the design requirements (6), the frequency response for the uncertain system varies. This variation affects:

- gain
- poles
- zeros

```
% lateral dynamics matrix assembly
run lateralDynamics;
% ...
% uncertain G & nominal G bode plot
bodemag(G, 'y', G.Nominal, 'k', {1e-3,1e+3});
```

Step Response

The step response brings to the **divergence** of the lateral dynamics motion. The uncertainties on data modify the position of **poles** and **zeros**. These changes **affect** the behaviour of the step response.



```
% lateral dynamics matrix assembly
run lateralDynamics;
% ...
% number of samples
n = 50;
% study time interval [s]
interval = 3;
% step response
step(usample(G, n), interval);
```

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P & PID Assembly

Constraints Setup

systune() Results

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Overall Steps

Aim: tune the **P** and **PID** blocks applying response **constraints**.
[1, Ch. 2] has been used as reference for the nominal design.
`feedbackDesign.m` → tunes the system. Program steps:

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 - $e_\varphi = S \varphi_0$
 - $\delta_{lat} = Q \varphi_0$

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 - **1st constraint** generation, computation of an alternative **sensitivity weight** function; sensitivityWeight.m
 - **2nd constraint** generation, **trial** and **error** based **control performance** weight function



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- **Requirements** setup
- Control **tuning** → K_φ , $K_{p,P}$, $K_{p,I}$ & $K_{p,D}$



P & PID Assembly

```

%% PID based loop (inner loop) assembly --> input ep = eInner
% PID controller definition -- roll rate
Rp = tunablePID('Rp', 'PID');
Rp.Kp.Value = 1e+2;
Rp.Ki.Value = 1e+2;
Rp.Kd.Value = 1e+2;
Rp.Tf.Value = 1e-3;
% input and output name declaration
Rp.u = 'e_{p}';
Rp.y = '\delta_{lat}';
% error at the inner node --> summation node
eInner = sumblk('e_{p} = p0 - p');
%% P based loop (outer loop) assembly --> input ephi = eOuter
% P controller definition -- roll angle
Rphi = tunablePID('Rphi', 'P');
Rphi.Kp.Value = 1e+2;
% input and output name declaration
Rphi.u = 'e_{\phi}';
Rphi.y = 'p0';
% error at the outer node --> summation node
eOuter = sumblk('e_{\phi} = \phi0 - \phi');

```

1st Constraint Setup

Target response:
$$\varphi = \frac{\omega_n^2}{s^2 + 2 \xi \omega_n s + \omega_n^2} \cdot \varphi_0 \quad (1)$$

There were some *problems* with `systune()` function if plugged in the **performance weight** related to (1)¹.

Sensitivity weight based on [1, Eq. (2.73)]:
$$W_P = \frac{\frac{s}{M} + \omega}{s + \omega A} \quad (2)$$

¹WSinv in `feedbackDesign.m`.



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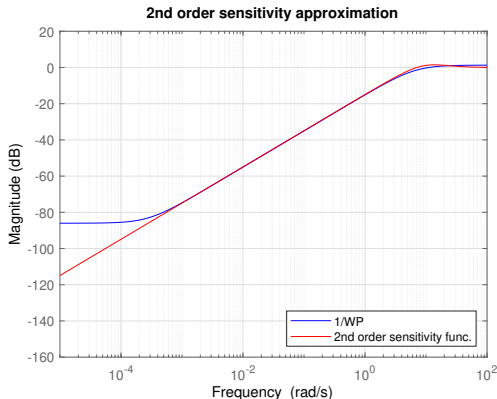
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`sensitivityWeight.m` **computes** M , ω & A values using:

- $-80db$ at **steady state**
- ω_n as reference for the **crossover** frequency
- $\xi \geq 0.9 \rightarrow [1]$ suggests $M \lesssim 1.185$

¹`WSinv` in `feedbackDesign.m`.

1st Constraint Approximation Result



sensitivityWeight.m
returns:

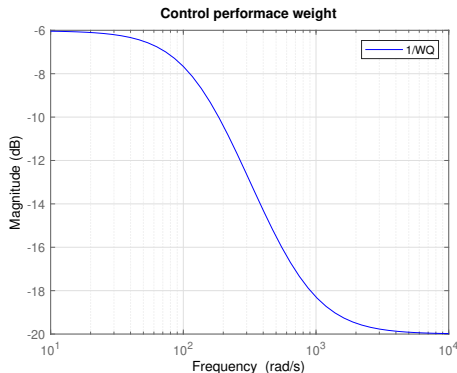
- $M = 1.16$
- $A = 0.00005$
- $\omega = 5.67$

2nd Constraint Setup

Target response: $|\varphi_0| \rightarrow |\delta_{lat}|$.

In this case a **pure trial** and **error** approach has been used for W_Q .

As before the **control performance weight** follows $W_Q = \frac{s}{s + \omega} \frac{1}{A}$.



Trial and **error** results:

- $M = 0.1$
- $A = 0.5$
- $\omega = 1.43e + 3$

Tuning Code

```
% connect() function
% input:
% --- phi0
% outputs:
% --- phi error (eOuter)
% --- delta
T0 = connect(G_nom, Rp, eInner, Rphi, eOuter, {'\phi0'}, {'e_{\phi}', '\delta_{lat}'});
% ...
% PERFORMANCE WEIGHT ASSEMBLY
% ...
% requirement vector assembly
req = [ TuningGoal.WeightedGain('\phi0', 'e_{\phi}', 1/WPinv, 1);
        TuningGoal.WeightedGain('\phi0', '\delta_{lat}', 1/WQinv, 1) ];
%% controllers tuning -- P & PID
opt = systuneOptions('RandomStart', nTest, 'SoftTol', 1e-7, 'Display', 'final');
% tuning control system
[T, J, -] = systune(T0, req, opt);
% getting values from the tuning results
Rp = T.blocks.Rp;
Rphi = T.blocks.Rphi;
```

Tuning Results

Stacking approach has been used for the **tuning**. Where:

$$\|W_P S\|_{\infty} < 1 \quad (3)$$

$$\|W_Q Q\|_{\infty} < 1 \quad (4)$$

$$\sigma = \sqrt{|W_P S|^2 + |W_Q Q|^2} < 1 \quad (5)$$

`systune()` has been used for the controllers **tuning**.
`systune()` works on T0 transfer function that links:

- $\varphi_0 \rightarrow e_{\varphi}$

- $\varphi_0 \rightarrow \delta_{lat}$

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- $\varphi_0 \rightarrow \delta_{lat}$

The **performance requirements** are stored in req object.

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`systeme()` works on T0 transfer function that links:

- $\varphi_0 \rightarrow e_{\varphi}$
- $\varphi_0 \rightarrow \delta_{lat}$

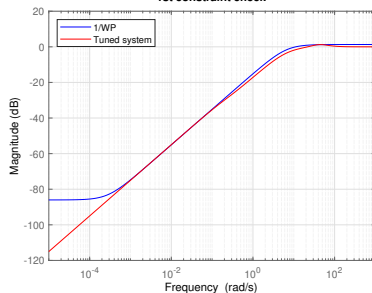
The **performance requirements** are stored in req object.
`systeme()` computes:

- $K_{\varphi} = 7.16$
- $K_{p,I} = 1.22$
- $K_{p,P} = 0.045$
- $K_{p,D} = -0.000157$

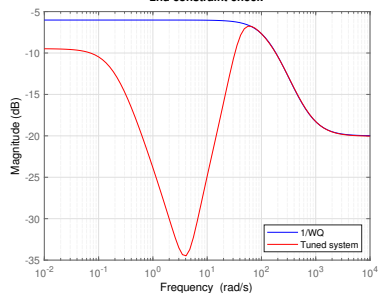


W_P^{-1} & W_Q^{-1} vs Tuning Results

1st constraint check



2nd constraint check

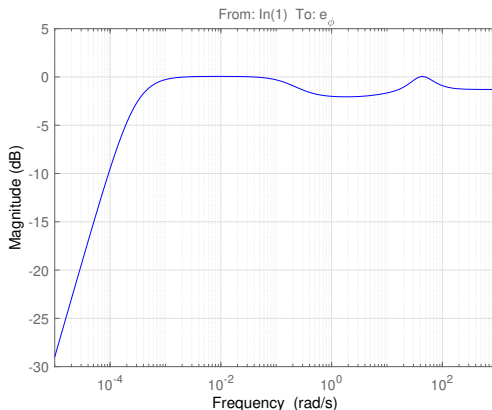


S Upperbound Check

The **performance weight** is expressed as a **boundary** transfer function:

$$\|W_P S\|_{\infty} < 1 \quad (6)$$

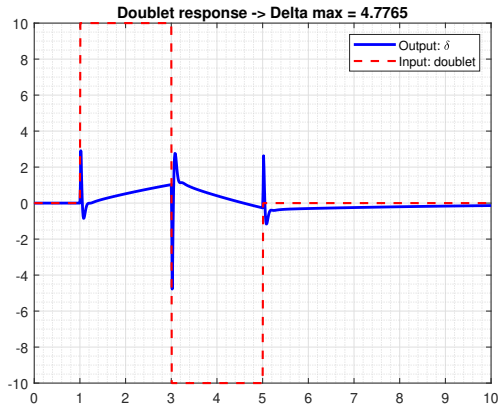
Sensitivity under the weight: $|WP * S|_{\text{Inf}} < 1$



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Doublet Response

From W_Q the closed loop system has been tuned with respect to $|\varphi_0| \rightarrow |\delta_{lat}|$.



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Uncertain Model

- The **deterministic uncertainties** can be modeled using the `ureal` function of MATLAB in the range of 3σ .

```
% stability derivatives
Y_v = ureal('Y_v',Y_v,'percentage',4.837*3); %1/s
L_v = ureal('L_v',L_v,'percentage',4.927*3); %rad s/m
% control derivatives
Y_d = ureal('Y_d',Y_d,'percentage',4.647*3); %m/s2
L_d = ureal('L_d',L_d,'percentage',2.762*3); %rad/s2
```

- Unlike in the previous case, here we are not going to pull out the nominal plant so the analysis will be carried out with the uncertain one.



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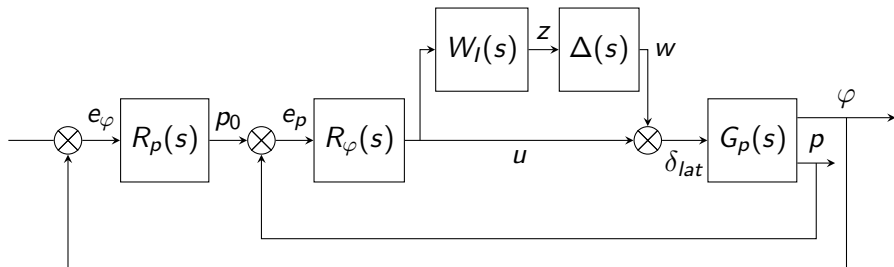
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$M - \Delta$ Construction



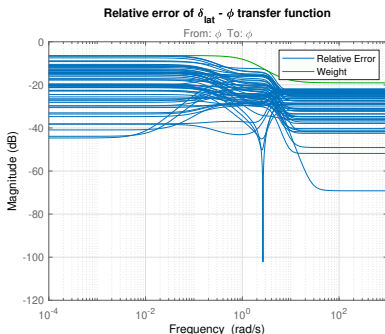
For our robustness analysis we use a system representation in which the uncertain perturbations are *pulled out* into a **complex** uncertain matrix $\Delta(s)$.

Type of representation: Input Multiplicative uncertainty.

- $W_l(s)$: Multiplicative weight
- $\Delta(s)$: Complex perturbation which is $\|\Delta(j\omega)\| \leq 1, \forall \omega$

Uncertain Weight

- The **relative error** is $\frac{G_{nom}-G_{unc}}{G_{nom}} = W_I(j\omega) \cdot \Delta(j\omega)$
- Knowing that $\Delta(j\omega) \leq 1$ gives us the upperbound: $\frac{G_{nom}-G_{unc}}{G_{nom}} \leq W_I(j\omega)$
- Consequently $W_I = 0.1114 s + \frac{0.664}{s+1.387}$



```
% from \delta_lat to \phi -> take the outer loop
G_phi = G(2,1);
% nominal plant
G_phi_nom = getNominal(G_phi);
% sampled uncertain plant
G_phi_smpl = usample(G_phi, 60);
% the weight order is one
[G_phi, Info] = uncover(G_phi_smpl, G_phi_nom, 1);
% nominal and uncertain plant relative error
Rel_e = (G_phi_nom - G_phi_smpl)/G_phi_nom;
% uncertain weight
W_I = minreal(tf(Info.W1));
```

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Robust Stability by $M - \Delta$

In order to study the robust stability:

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In order to study the robust stability:

- 1 The feedback system is represented in the $M - \Delta$ form

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- 2 The magnitude of M has to be found
- 3 The robust stability needs to be checked

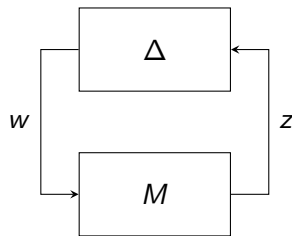
Robust Stability by $M - \Delta$

In order to study the robust stability:

- 1 The feedback system is represented in the $M - \Delta$ form
- 2 The magnitude of M has to be found
- 3 The robust stability needs to be checked

By applying the **Nyquist stability** criterion we assume that $M = W_l F(s)$ and Δ are stable. The Nyquist stability condition then determines RS if and only if the **loop transfer function** M does not encircle $-1, \forall \Delta$, [1].

$$\text{RS} \Leftrightarrow |1 + M\Delta| > 0, \forall \omega, \forall |\Delta| < 1$$
$$\|M(j\omega)\| \leq 1, \forall \omega, \forall |\Delta| < 1, \forall \omega$$



MATLAB Implementation

```

W_I.u = '\delta_{lat}'; % weight input
W_I.y = 'z'; % weight output
G_n = getNominal(G); % nominal plant
G_n.u = 'u_delta_{lat}'; % input of nominal plant
Sum_u = sumblk('u_delta_{lat} = \delta_{lat} + w'); % multiplicative sum
eOuter_new = sumblk('e_{\phi} = - \phi'); % not considering setpoint in sum block
M = connect(Rphi, eInner, Rp, W_I, Sum_u, G_n, eOuter_new, {'w'}, {'z'}); % M TF
M_s = minreal(M); % simplified M TF
get(M_s)

```

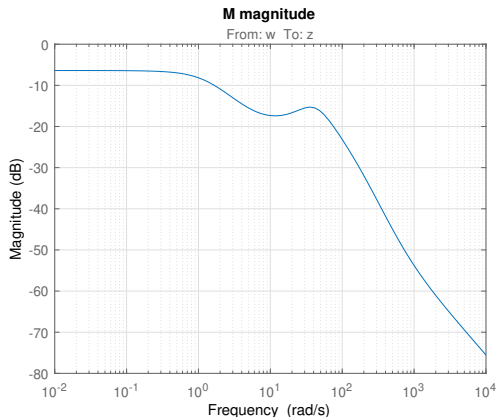
Where M is constituted by:

$$A = \begin{bmatrix} -1.387 & 0 & -0.014 & -0.099 & 1.227 & -0.032 \\ 0 & -0.264 & -0.133 & 8.859 & 11.745 & -0.305 \\ 0 & -7.349 & -14.975 & -107.296 & 1324.926 & -34.468 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & -1 & -7.164 & 0 & 0 \\ 0 & 0 & 203.224 & 1456.077 & 0 & -203.224 \end{bmatrix}, B = \begin{bmatrix} 0 \\ 9.568 \\ 1079.339 \\ 0 \\ 0 \\ 0 \end{bmatrix}, \\
 C = [0.509 \quad 0 \quad -0.0015 \quad -0.011 \quad 0.136], D = 0$$



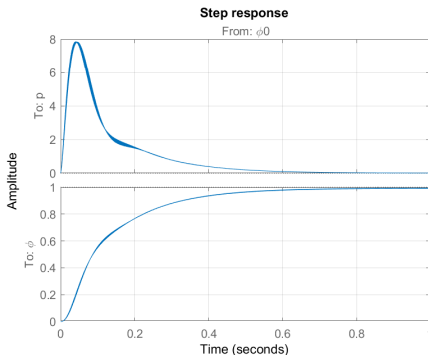
Magnitude of M

By taking a look at the magnitude of M , it can be stated that **robust stability** was achieved.

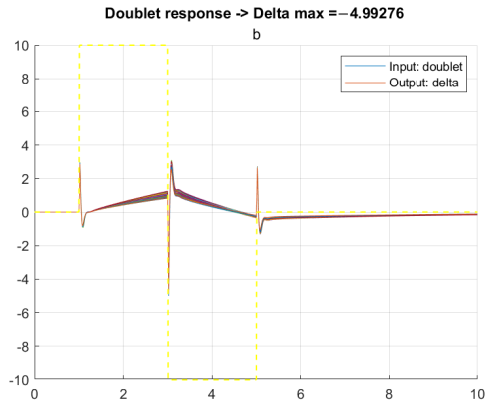


Time Domain Result

- The output reaches its desired value after a short period which confirms that robust stability is satisfied
- Roll rate** reaches **zero** at steady state while also considering uncertainties



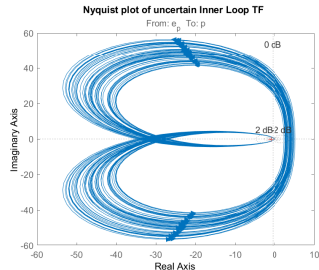
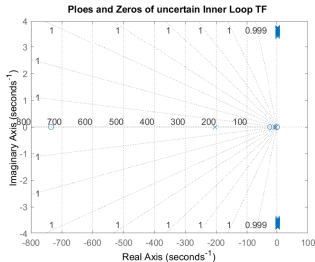
Uncertain Doublet Response



Inner Loop Stability

Nyquist Stability Criterion has been used to ensure the Inner loop stability.

- No. of RHP poles: **2 RHP complex conjugate poles** near zero
- The -1 points needs to be encircled **twice**



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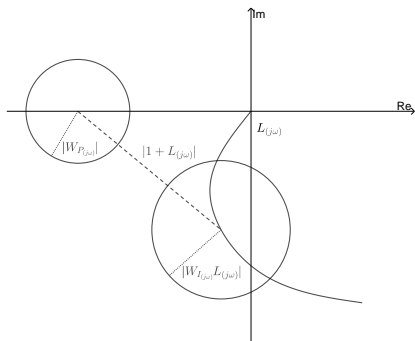
6 References



POLITECNICO
MILANO 1863

Robust Performance – I

Geometric derivation



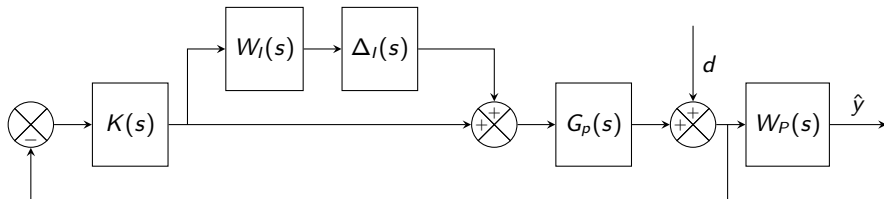
$$RP \Leftrightarrow |W_P| + |W_I L| \leq |1 + L|, \forall \omega$$

$$RP \Leftrightarrow \left| \frac{W_P}{1 + L} \right| + \left| \frac{W_I L}{1 + L} \right| \leq 1, \forall \omega$$

$$RP \Leftrightarrow \max_{\omega} (|W_P S| + |W_I F|) \leq 1$$



Robust Performance – II



For RP we require the performance condition to be satisfied for all possible plants including the worst uncertain case:

Algebraic derivation of RP condition:

$$\text{RP} \Leftrightarrow \max_{S_P} |W_P S_P| \leq 1, \forall \omega \quad (7)$$

In which $S_P = \frac{1}{I+L_P} = \frac{1}{1+L+W_I L \Delta_I}$

$$\text{RP} \Leftrightarrow \max_{S_P} |W_P S_P| = \frac{|W_P|}{|1+L|} - |W_I L| \quad (8)$$

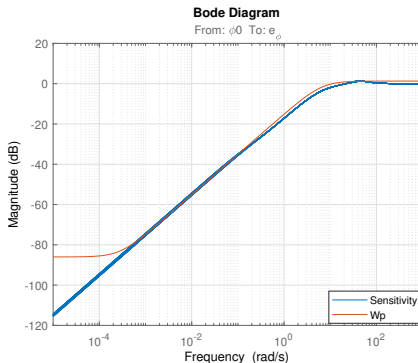
Finally, the following result is obtained:

$$\boxed{\text{RP} \Leftrightarrow \max_{\omega} (|W_P S| + |W_I F|) \leq 1} \quad (9)$$

Robust Performance – III

Considering worst case weighted sensitivity:

$$\text{RP} \Leftrightarrow \max_{S_p} \|W_p S_p\| < 1, \forall \omega \quad (10)$$

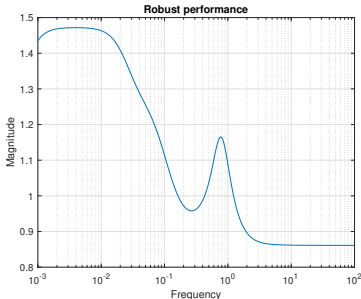


Robust Performance – IV

The RP condition in **SISO** case can be closely approximated by mixed sensitivity H_{inf} condition:

$$\text{RP} \Leftrightarrow \max_{\omega} \sqrt{|W_p S_p|^2 + |W_l F|^2} < 1 \quad (11)$$

and it can be done in **MIMO** case by structured singular value as we will see later.



We can simply define an index to analyse RP by summing up the NP and RS inequalities

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$M - \Delta$ Construction

- Separate nominal and uncertain parts of the uncertain parameters as we did before, while here δ_i s are **real** numbers such that $||\delta_i|| \leq 1$:

$$\begin{aligned}
 & \bullet Y_v = Y_{v0} (1 + r_{Y_v} \delta_1); & \bullet Y_\delta = Y_{\delta0} (1 + r_{Y_\delta} \delta_3); \\
 & \bullet L_v = L_{v0} (1 + r_{L_v} \delta_2); & \bullet L_\delta = L_{\delta0} (1 + r_{L_\delta} \delta_4);
 \end{aligned}$$

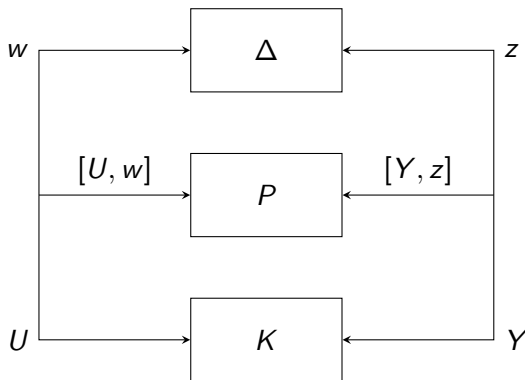
where $Y_{i0} = \frac{Y_m + Y_M}{2}$ and $r_i = \frac{Y_m - Y_M}{Y_m + Y_M}$

- Update the state space coefficient Matrices:

$$\mathbf{A} = \mathbf{A}_{nom} + \begin{bmatrix} r_{Y_v} \\ 0 \\ 0 \end{bmatrix} \delta_1 \begin{bmatrix} 1 & 0 & 0 \end{bmatrix} + \begin{bmatrix} 0 \\ r_{L_v} \\ 0 \end{bmatrix} \delta_2 \begin{bmatrix} 0 & 1 & 0 \end{bmatrix}$$

$$\mathbf{B} = \mathbf{B}_{nom} + \begin{bmatrix} r_{Y_\delta} \\ 0 \\ 0 \end{bmatrix} \delta_3 + \begin{bmatrix} 0 \\ r_{L_\delta} \\ 0 \end{bmatrix} \delta_4$$

μ Analysis – Structured Singular Value



- $\Delta(s)$: $\text{diag} \Delta_i$
- P : Nominal Plant Matrix (5x6)
- K : Controller Matrix

P and K together construct the M which finally gives us the well known $M - \Delta$ model.



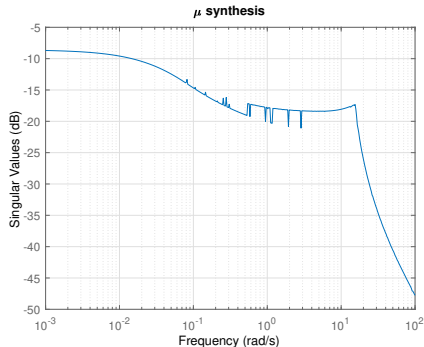
μ Analysis – Code

```
% separates the uncertain part from certain part
[P, Delta, BlkStruct] = lftdata(G);
P.u = {'w1','w2','w3','w4','\delta_{lat}'}; % inputs of P
P.y = {'z1','z2','z3','z4','p','\phi'}; % outputs of P
eOuter_new = sumblk('e_{\phi} = - \phi');

% computing M for mu synthesis
M_mu = connect(Rphi, eInner, Rp, P, eOuter_new, ...
    {'w1','w2','w3','w4'}, ...
    {'z1','z2','z3','z4'});
```



Stability Analysis

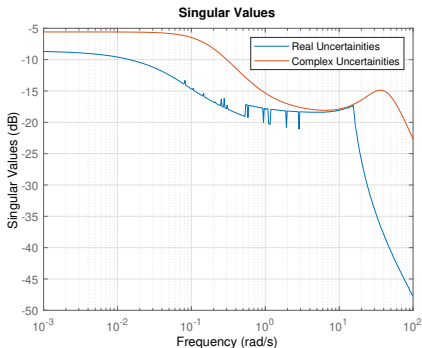


MATLAB code:

```
omega = logspace(-3, 2, 500);
bounds = mussv(frd(M_mu, omega), [-1 0; -1 0; -1 0; -1 0]);
```

- Assuming having each individual perturbation to be stable
- **Condition for RS in μ synthesis:**
- RS if $\bar{\sigma}(M(j\omega)) < 1, \forall \omega$

Uncertainty Impact on μ Stability



Effect of Real uncertainty:

- Stability margin gets **tighter** when using real uncertainties
- We are less conservative

MATLAB code:

```
bounds      = mussv(frd(M_mu,omega), [-1 0; -1 0; -1 0; -1 0]); % real uncertainties
bounds_com  = mussv(frd(M_mu,omega), [ 1 0;  1 0;  1 0;  1 0]); % complex uncertainties
```

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Monte-Carlo Simulation

Monte-Carlo simulation is a technique used to study how a model responds to randomly generated inputs.

²In monteCarlo.m: 500 times.

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- 1 Randomly generate N inputs.
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Monte-Carlo simulation is a technique used to study how a model responds to randomly generated inputs.

- 1 Randomly generate N inputs.
- 2 Run a simulation for each of the N inputs².
- 3 Aggregate and assess the outputs from the simulations.

Common measures include the mean value of an output, the distribution of output values, and the minimum or maximum output value.

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- ② Run a simulation for each of the N inputs².
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Common measures include the mean value of an output, the distribution of output values, and the minimum or maximum output value.

MATLAB code:

% covariance

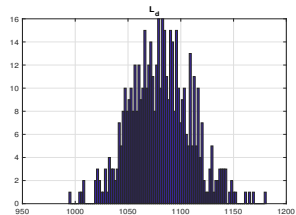
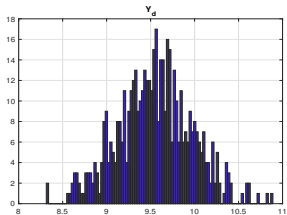
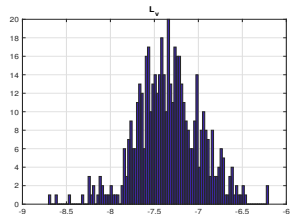
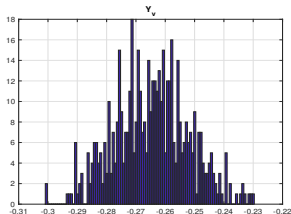
```
sigma_Yv = ((4.837/100)*abs(Y_v))^2;
sigma_Lv = ((4.927/100)*abs(L_v))^2;
sigma_Yd = ((4.647/100)*abs(Y_d))^2;
sigma_Ld = ((2.762/100)*abs(L_d))^2;
```

% gaussian distribution

```
Y_v_gd = gmdistribution(Y_v, sigma_Yv);
L_v_gd = gmdistribution(L_v, sigma_Lv);
Y_d_gd = gmdistribution(Y_d, sigma_Yd);
L_d_gd = gmdistribution(L_d, sigma_Ld);
```

²In monteCarlo.m: 500 times.

Gaussian Distribution of Uncertain Parameters



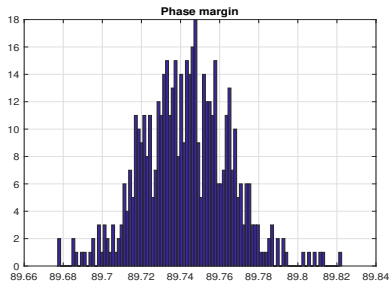
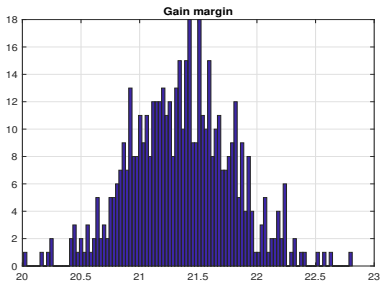
Simulation Results for System Margins

Nominal Gain Margin (G_m)

21.3451

Nominal Phase Margin (P_m)

89.7431



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- [1] Sigurd Skogestad and Ian Postlethwaite.
Multivariable feedback control: analysis and design.
Citeseer, 2007.

Thank you!

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