

Geometry and Mechanics

Transformation notation
 A^*B , AB , A^*B^* , A_B , A_B^*

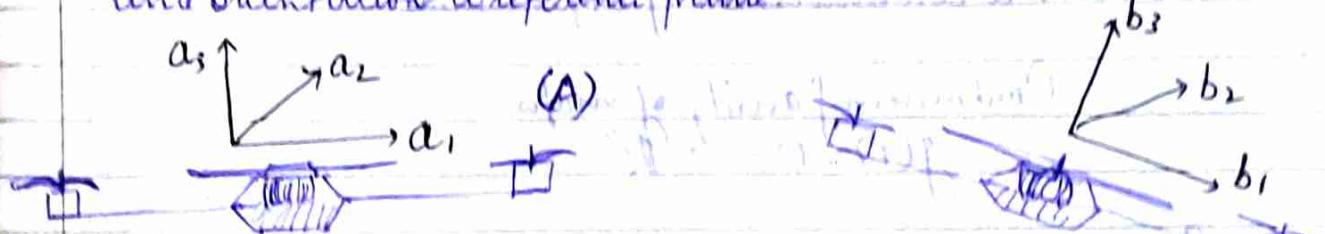
Quadrilater Kinematics

A_B , R_B

Rigid Body Transformations

g_{AB} , h_{AB}

Reference frame — we associate with each position and orientation a reference frame

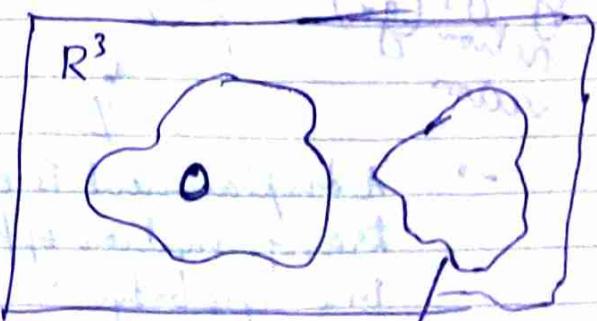


- In reference frame A, we find three linearly independent basis vectors a_1, a_2 & a_3 . Here, they are mutually ~~1~~ orthogonal. While they don't have to be, it's convenient to choose them to be mutually ~~1~~ orthogonal.
- The key idea is that they must ~~be~~ be linearly independent.
- We can write any vector as a linear combination of the basis vectors in either frame.

$$v = v_1 a_1 + v_2 a_2 + v_3 a_3 \text{ (in frame A)} \rightarrow \text{Similarly for frame B}$$

Rigid Body Displacement

Object $O \subset \mathbb{R}^3$ (subset of \mathbb{R}^3)



- A rigid body displacement is nothing but a map from this collection of points in the object to its physical manifestation in \mathbb{R}^3 (Real Space).

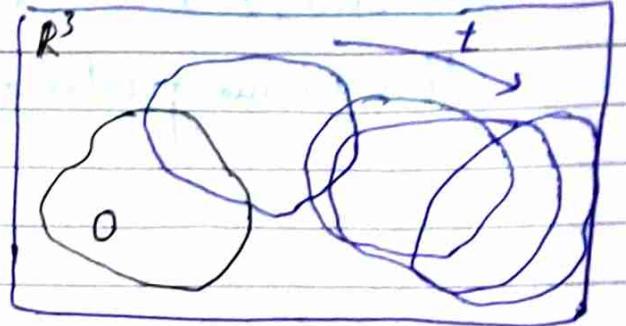
$$g: O \rightarrow \mathbb{R}^3$$

Same object O but with a diff pose/orient meaning the matter way

- As time changes, this object may occupy different positions and orientations.
- Accordingly, we have different rigid body displacements.

What is rigid body motion?

- As time ~~passes~~ changes, we have a continuous family of maps, so g (which is a displacement) is now parametrized by time.
- As time changes, we have the collection of points in O , moving from one position & orientation to another & soon. This is a continuous set of displacements.



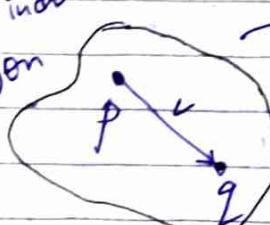
Continuous family of maps
 $g(t) : O \rightarrow \mathbb{R}^3$

Eg. this is what we see in a quadrotor, starting off from a horizontal position, moving to another position, decelerating, changing its orientation, changing the dir. of thrust & then reversing the dir. of the thrust by pitch back & then slowing down to the goal position.

Each of these snapshots is a displacement. This sequence of displacements represents a continuous family of displacements.

A single point on the rigid body p

Transformation (g) of points induces an action (g_*) on vectors.



Within this rigid body is displaced, the point p gets

displaced to a new point, $g(p)$

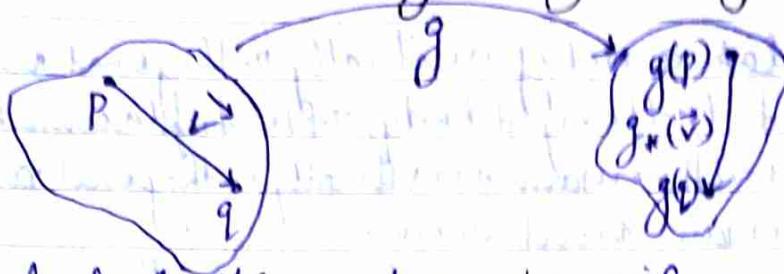
- A displacement is essentially a transformation of points. There are ∞ points in a rigid body.
- If we have a second point q , the same displacement will take q & move it into a new point $g(q)$.
- Every point defines a vector.

Since g maps p to $g(p)$ and $q \rightarrow g(q)$, it also moves the vector v to a new vector, $g^*(v)$.

So, displacement g reduces a map on vector.

★ (displacement acts on points, but g^* acts on vectors)

What makes the mat, g. a rigid body displace?.



Two properties the map must satisfy -

① the distance betⁿ any pair of points remains unchanged in a rigid body displacement, i.e.,

$$\text{(Length must be preserved)} \quad \|g(p) - g(q)\| = \|p - q\| \quad \text{(def'n of the word "rigid")}$$

(after the displacement) (before the displacement)

11 (related to the cross products of vectors that are attached to the rigid body)

Let's choose a third point 'q' & join to a second vector going from $P \rightarrow q$

$$\cancel{\text{(cross products are reversed) }} g_*(\vec{v}) \times g_*(\vec{w}) = g_*(\vec{v} \times \vec{w})$$

→ for a rigid body displacement,

→ orthogonal vectors are mapped to orthogonal vectors.

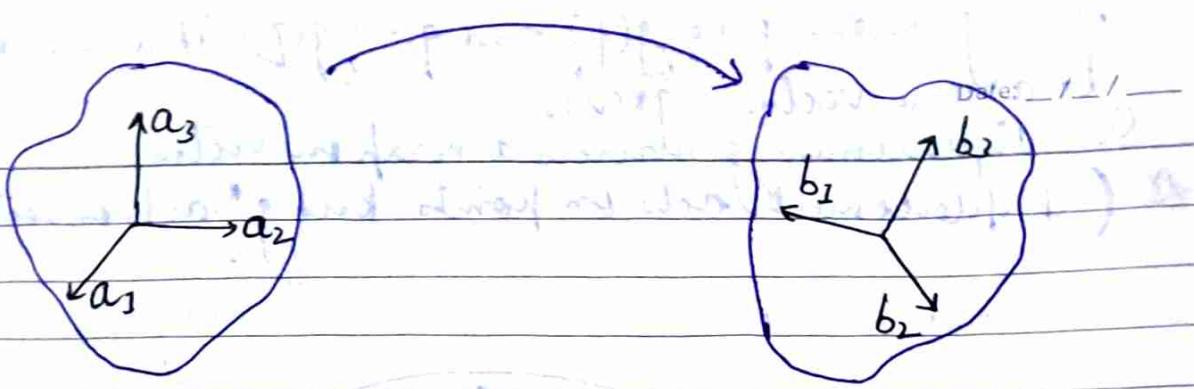
$\rightarrow g^*$ preserves inner products

$$g_{\star}(\vec{v}) \circ g_{\star}(\vec{w}) = \vec{v} \cdot \vec{w}$$

~~(before displacement)~~ (before displacement)

(After displacement)

Cross product remains the same whether we do it before the displacement or after the displacement, if the displacement is rigid.

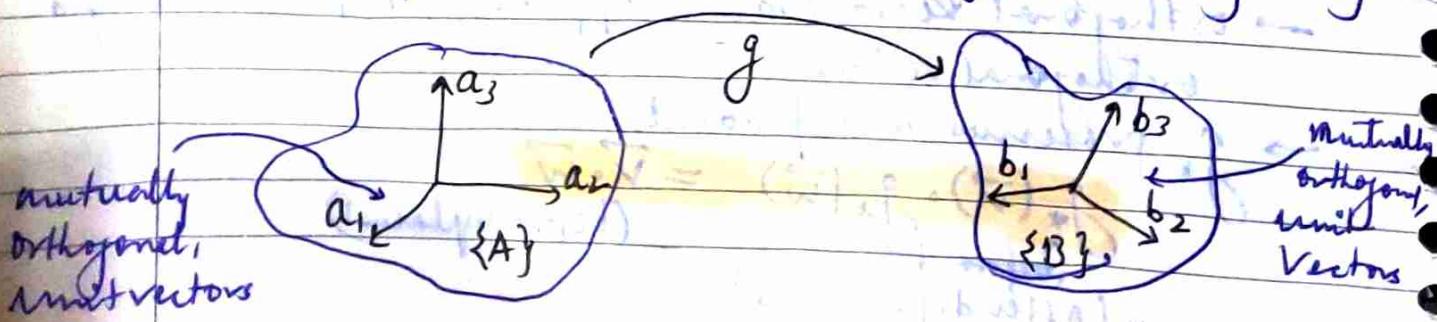


- If we take a set of mutually orthogonal unit vectors attached to the rigid body, after displacement, those vectors will remain mutually orthogonal and they will remain unit vectors.
- In summary, rigid body transformations or rigid body displacements satisfy two important properties. They preserve lengths and they preserve cross products. (map preserves lengths) (cross products are preserved by the induced map)

Note: Rigid body displacements and rigid body transformation are not interchangeably. There is one important semantic difference.

Transformations generally used to describe relationship between reference frames attached to different rigid bodies. While displacements describe relationships between two positions and orientation of a frame attached to a displaced rigid body.

Calculations :- (Assumption) we have mutually orthogonal unit vectors attached to every rigid body. If it's a transformation, we are referring to two different rigid bodies. If it's a displacement, it's two distinct positions and orientation of the same rigid body.



Mutually orthogonal unit vectors in one frame as a linear combination of mutually orthogonal unit vectors in the other frame.

$$\hat{b}_1 = R_{11} \hat{a}_1 + R_{12} \hat{a}_2 + R_{13} \hat{a}_3$$

$$\hat{b}_2 = R_{21} \hat{a}_1 + R_{22} \hat{a}_2 + R_{23} \hat{a}_3$$

$$\hat{b}_3 = R_{31} \hat{a}_1 + R_{32} \hat{a}_2 + R_{33} \hat{a}_3$$



(rotation matrix) $R = \begin{bmatrix} R_{11} & R_{12} & R_{13} \\ R_{21} & R_{22} & R_{23} \\ R_{31} & R_{32} & R_{33} \end{bmatrix}$

Properties of a Rotation Matrix

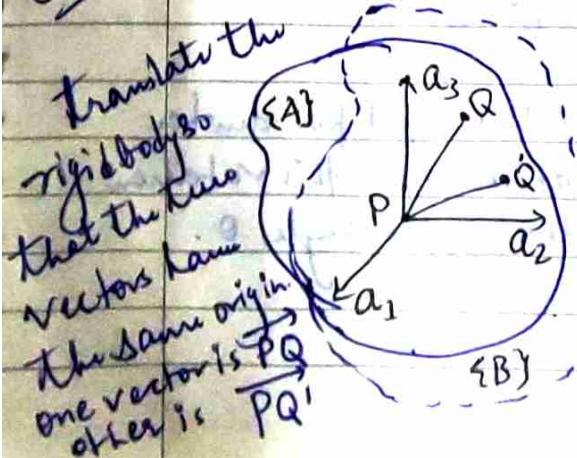
- I Orthogonal $\Rightarrow RR' = I = R'R$
- II Special orthogonal $\Rightarrow |R| = +1$
- III Closed under multiplication \rightarrow the product of any two rotation matrices is another rotation matrix.
- IV The inverse of a rotation matrix is also a rotation matrix.

Structure of a Rotation Matrix

two distinct past orientations of the same rigid body

$\vec{PQ} = q_1 \hat{a}_1 + q_2 \hat{a}_2 + q_3 \hat{a}_3$

$\vec{PQ} = q_1 \hat{b}_1 + q_2 \hat{b}_2 + q_3 \hat{b}_3$



$$\vec{PQ} = q_1 \hat{a}_1 + q_2 \hat{a}_2 + q_3 \hat{a}_3$$

$$\vec{PQ}' = q'_1 \hat{a}_1 + q'_2 \hat{a}_2 + q'_3 \hat{a}_3$$

$$\begin{bmatrix} q'_1 \\ q'_2 \\ q'_3 \end{bmatrix} = \begin{bmatrix} ? & ? & ? \\ ? & ? & ? \\ ? & ? & ? \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \\ q_3 \end{bmatrix}$$

How to write (q'_1, q'_2, q'_3) as a function of (q_1, q_2, q_3)

OR

$$\begin{bmatrix} q_1 \\ q_2 \\ q_3 \end{bmatrix} = \begin{bmatrix} ? & ? & ? \\ ? & ? & ? \\ ? & ? & ? \end{bmatrix} \begin{bmatrix} q_1' \\ q_2' \\ q_3' \end{bmatrix}$$

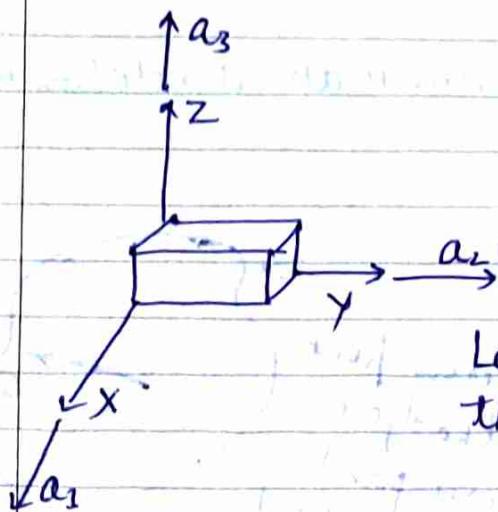
↓
Rotation matrix.

i.e., $\begin{bmatrix} q_1 \\ q_2 \\ q_3 \end{bmatrix} = \begin{bmatrix} R_{11} & R_{12} & R_{13} \\ R_{21} & R_{22} & R_{23} \\ R_{31} & R_{32} & R_{33} \end{bmatrix} \begin{bmatrix} q_1' \\ q_2' \\ q_3' \end{bmatrix}$

(Rotation matrix)

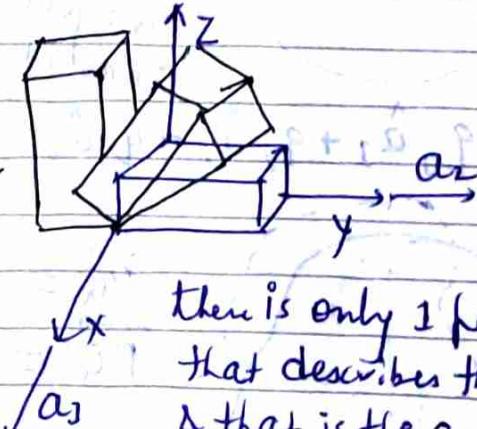
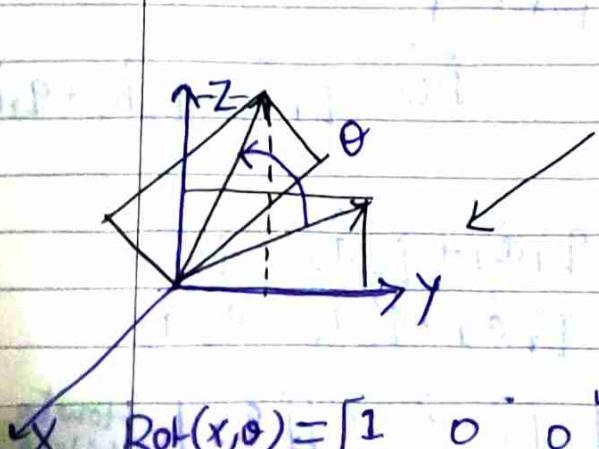
→ If we know how to write mutually orthogonal unit vectors in one frame as a function of the other set of vectors (we can do this by calculating Rotation matrix). The same matrix tells us how to transform vectors in one frame to another frame.

e.g. Rotation - Rotation about the x-axis through θ .



Consider this rigid body - a rectangular prism, whose axes are aligned with x, y & z-axis or the a_1, a_2, a_3 unit vectors.

Let's rotate this rigid body about the x-axis, through an angle, θ .

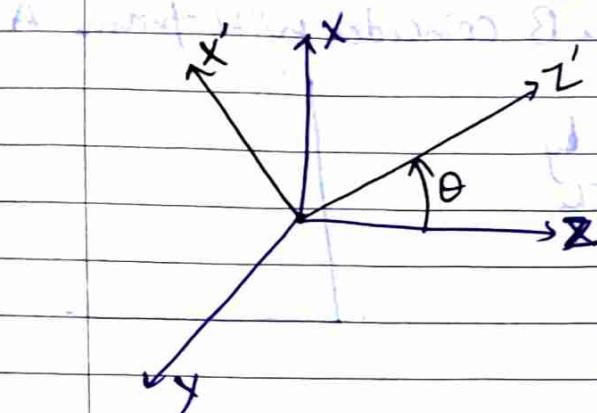


there is only 1 parameter that describes this rotation & that is the angle, θ .

$$\text{Rot}(x, \theta) = \begin{bmatrix} 1 & 0 & 0 \\ 0 & \cos \theta & -\sin \theta \\ 0 & \sin \theta & \cos \theta \end{bmatrix}$$

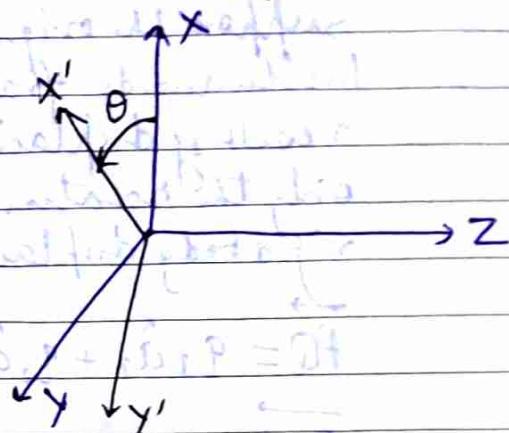
Rotation about the y-axis through θ

$$\text{Rot}(y, \theta) = \begin{bmatrix} \cos \theta & 0 & \sin \theta \\ 0 & 1 & 0 \\ -\sin \theta & 0 & \cos \theta \end{bmatrix}$$

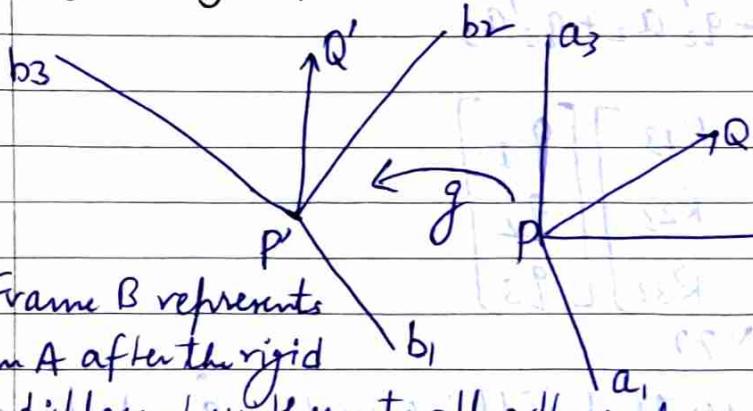


Rot. about z-axis through θ

$$\text{Rot}(z, \theta) = \begin{bmatrix} \cos \theta & -\sin \theta & 0 \\ \sin \theta & \cos \theta & 0 \\ 0 & 0 & 1 \end{bmatrix}$$



Ex- Rigid Body Displacement



Let Frame B represents Frame A after the rigid body displacement with mutually orthogonal unit vectors $\vec{b}_1, \vec{b}_2, \vec{b}_3$ with origin at P. Point Q after the rigid body displacement is Q'.

$$\vec{PQ}' = q_1 \vec{b}_1 + q_2 \vec{b}_2 + q_3 \vec{b}_3$$

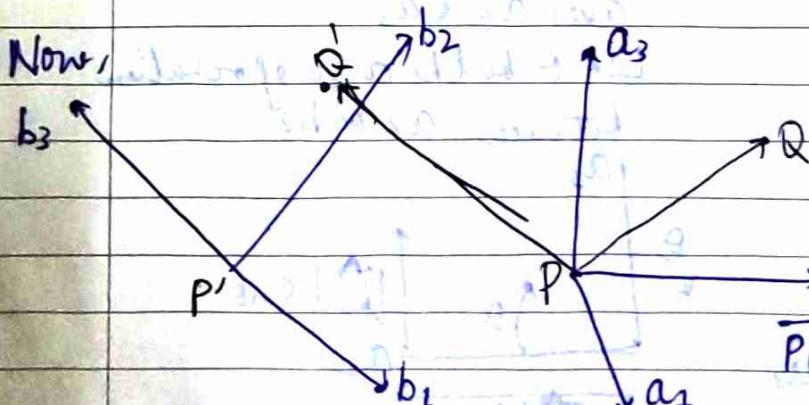
Let Frame A with mutually orthogonal unit vectors $\vec{a}_1, \vec{a}_2, \vec{a}_3$ with origin P attached to a rigid body.

Let Q be a point on this rigid body.

\vec{PQ} represents position of point Q relative to frame A.

Suppose the body undergoes a rigid body displacement 'g'.

$$\vec{PQ} = q_1 \vec{a}_1 + q_2 \vec{a}_2 + q_3 \vec{a}_3$$



\vec{PQ}' gives the position of the displaced point Q' relative to frame A.

$$\begin{bmatrix} q_1' \\ q_2' \\ q_3' \end{bmatrix} = R \begin{bmatrix} q_1 \\ q_2 \\ q_3 \end{bmatrix} + d$$

Rotation between frame A and frame B

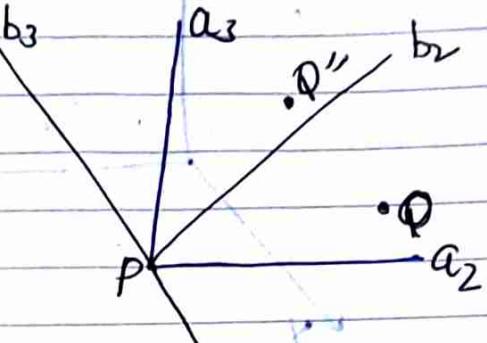
Translate
 $\{B\}_{\text{so}}$
 reference
 frame
 share an
 origin

Suppose the origin of frame B coincides with frame A
 In other words, frame B is a result of displacing frame A by only the rotation portion of the rigid body displacement.

$$\overrightarrow{PQ} = q_1 \hat{a}_1 + q_2 \hat{a}_2 + q_3 \hat{a}_3$$

$$\overrightarrow{PQ''} = q_1 \hat{b}_1 + q_2 \hat{b}_2 + q_3 \hat{b}_3$$

$$= q_1'' \hat{a}_1 + q_2'' \hat{a}_2 + q_3'' \hat{a}_3$$



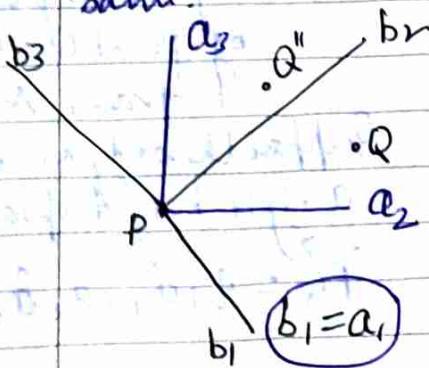
\overrightarrow{PQ} is the position vector of Q in $\{A\}$

$\overrightarrow{PQ''}$ is the position vector of Q after the rotation in frame B / $\{B\}$

$$\begin{bmatrix} q_1'' \\ q_2'' \\ q_3'' \end{bmatrix} = \begin{bmatrix} R_{11} & R_{12} & R_{13} \\ R_{21} & R_{22} & R_{23} \\ R_{31} & R_{32} & R_{33} \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \\ q_3 \end{bmatrix}$$

??

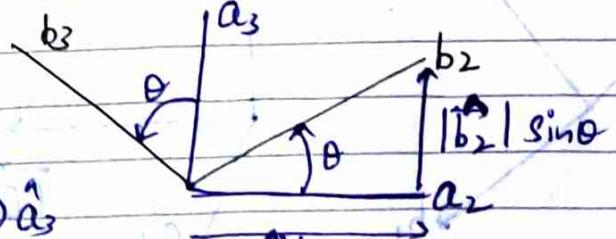
we can easily see that, in this example, \hat{a}_3 & \hat{b}_3 are the same.



⇒ This means axes b_2 & b_3 are in the same plane as the axes a_2 & a_3 .

Let θ be the angle of rotation between a_2 & b_2 .

$$\begin{aligned} \hat{b}_3 &= -(\hat{b}_3 | \sin \theta) \hat{a}_2 + (\hat{b}_3 | \cos \theta) \hat{a}_3 \\ &= -(\sin \theta) \hat{a}_2 + (\cos \theta) \hat{a}_3 \end{aligned}$$



$$\begin{aligned} \hat{b}_2 &= (\hat{b}_2 | \cos \theta) \hat{a}_2 + (\hat{b}_2 | \sin \theta) \hat{a}_3 \\ &= (\cos \theta) \hat{a}_2 + (\sin \theta) \hat{a}_3 \end{aligned}$$

$|b_1| = |b_2| = |b_3| = 1$ (unit vectors)

$$\overrightarrow{PQ}'' = q_1 \hat{b}_1 + q_2 \hat{b}_2 + q_3 \hat{b}_3$$

$$= q_1'' \hat{a}_1 + q_2'' \hat{a}_2 + q_3'' \hat{a}_3$$

Date _____

$$\overrightarrow{PQ}'' = q_1 (\hat{a}_1) + q_2 [a_2 \cos \theta + a_3 \sin \theta] + q_3 [-a_2 \sin \theta + a_3 \cos \theta]$$

$$= q_1 \hat{a}_1 + (q_2 \cos \theta - q_3 \sin \theta) \hat{a}_2 + (q_2 \sin \theta + q_3 \cos \theta) \hat{a}_3$$

$$\overrightarrow{PQ}'' = q_1'' \hat{a}_1 + q_2'' \hat{a}_2 + q_3'' \hat{a}_3$$

$$\overrightarrow{PQ}' = q_1 \hat{a}_1 + (q_2 \cos \theta - q_3 \sin \theta) \hat{a}_2 + (q_2 \sin \theta + q_3 \cos \theta) \hat{a}_3$$

$$\therefore q_1'' = q_1$$

$$q_2'' = q_2 \cos \theta - q_3 \sin \theta$$

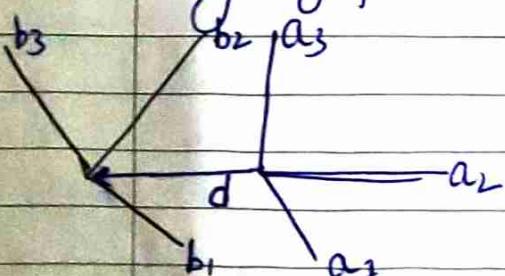
$$q_3'' = q_2 \sin \theta + q_3 \cos \theta$$

$$\text{So, } \begin{bmatrix} q_1'' \\ q_2'' \\ q_3'' \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & \cos \theta & -\sin \theta \\ 0 & \sin \theta & \cos \theta \end{bmatrix} \begin{bmatrix} q_1 \\ q_2 \\ q_3 \end{bmatrix}$$

$$\therefore R = \begin{bmatrix} 1 & 0 & 0 \\ 0 & \cos \theta & -\sin \theta \\ 0 & \sin \theta & \cos \theta \end{bmatrix} = \text{Rot}(x, \theta)$$

Translation between frame A and frame B

Let d be the ^{position} vector from the origin of frame A to the origin of frame B, expressed in terms of frame A.



$$\text{eg. } \overrightarrow{d} = 1 \hat{a}_1 - 3 \hat{a}_2 + 1 \hat{a}_3$$

$$= \begin{bmatrix} 1 \\ -3 \\ 1 \end{bmatrix}$$

→ we can characterize a rigid-body displacement with a rotation matrix and translation vector.

$$\text{Let } \theta = \pi/4 \Rightarrow R = \begin{bmatrix} 1 & 0 & 0 \\ 0 & \frac{1}{\sqrt{2}} & \frac{1}{\sqrt{2}} \\ 0 & \frac{1}{\sqrt{2}} & \frac{1}{\sqrt{2}} \end{bmatrix}$$

$$\begin{bmatrix} q_1' \\ q_2' \\ q_3' \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & \frac{1}{\sqrt{2}} & \frac{1}{\sqrt{2}} \\ 0 & \frac{1}{\sqrt{2}} & \frac{1}{\sqrt{2}} \end{bmatrix} \begin{bmatrix} 1 \\ 2 \\ 1 \end{bmatrix} + \begin{bmatrix} 1 \\ -3 \\ 1 \end{bmatrix}$$

Components
of \vec{PQ} in
frame A' (A)

$$= \begin{bmatrix} 2 \\ -2.29 \\ 3.12 \end{bmatrix}$$

Suppose we know that
the position vector \vec{PQ}
is given by components

Qualitatively, we can see that
point Q' relative to ~~point~~ {A'} is in the
position \vec{a}_3 , negative \vec{a}_2 & positive \vec{a}_3 direction

→ If we know the components of \vec{PQ}' , we can
invert the previous equation to find the components of \vec{PQ} .

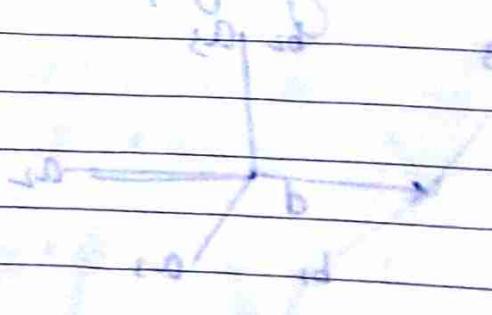
$$\begin{bmatrix} q_1 \\ q_2 \\ q_3 \end{bmatrix} = R^T \left(\begin{bmatrix} q_1' \\ q_2' \\ q_3' \end{bmatrix} - \vec{d} \right)$$

∴ A unit vector along the direction of \vec{PQ} is

∴ unit vector along the direction of \vec{PQ} is

$$\vec{PQ} = \vec{PQ}' + \vec{d}$$

$$\begin{bmatrix} q_1 \\ q_2 \\ q_3 \end{bmatrix} = \vec{PQ}' + \vec{d}$$



∴ unit vector along the direction of \vec{PQ} is

Rotation

Special Orthogonal Matrices

Date: 1/1/2023

$$SO(3) = \{ R \in \mathbb{R}^{3 \times 3} \mid R^T R = R R^T = I, \det R = 1 \}$$

→ Special orthogonal group in 3 dimensions
(the group of rotations is called $SO(3)$)

Coordinates for $SO(3)$

- ① Rotation matrices
- ② Euler angles
- ③ Axis angle parametrization
(that explicitly describes the axis of rotation & angle of rotation)
- ④ Exponential Coordinates
- ⑤ Quaternions

Coordinates for a Sphere

→ Parameterize using a set of local coordinate charts
(latitude and longitude)

- Q) Given any point on the Earth surface, is there a unique combination of latitudes and longitudes that describe that point?
→ No, if we consider the two poles. The poles have a unique latitude but the longitudes are not well-defined.

Function - A relation that assigns each element in a set of inputs, X called the domain to exactly one element in a set of outputs, Y called the codomain (or range).

$$f: X \rightarrow Y$$

One-to-One (injective) - $\forall a, b \in X$, if $f(a) = f(b)$, then $a = b$.
No two inputs from the domain will map to the same output in the codomain.

Onto (surjective) - $\forall y \in Y$, $\exists x \in X$ such that $f(x) = y$.
Every output in the codomain has an input in the domain that maps to it.

→ If all imaginary lines parallel to x -axis intersect the graph at most once, then the function is 1-1.

$$y: f: \mathbb{R} \rightarrow \mathbb{R} \text{ s.t. } f(x) = \ln(x)$$

This function is not onto

for any $y \leq 0$, there is no x such that

$$\ln(x) = y$$

But, $f: \mathbb{R} \rightarrow (0, \infty)$ s.t. $f(x) = \ln(x) \rightarrow$ is onto (as specified codomain)

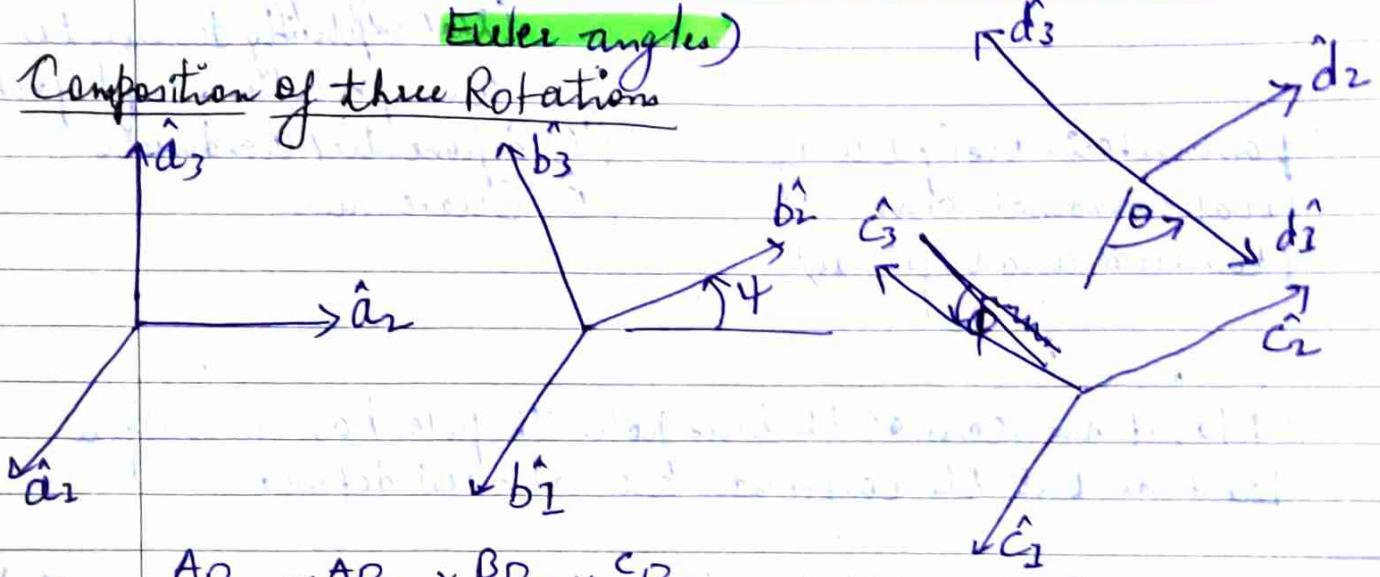
What is the minimum number of coordinates we need to parameterise $SO(3)$?

Date: 1st Jan 2022

$$SO(3) = \{R \in \mathbb{R}^{3 \times 3} \mid R^T R = R R^T = I, \det R = 1\}$$

Euler Angles (Euler showed that 3 coordinates are necessary to describe a general rotation and these coordinates are called the Euler angles)

Composition of three Rotations



$${}^A R_D = {}^A R_B \times {}^B R_C \times {}^C R_D$$

$$[{}^A R_D = \text{Rot}(x, \psi) \times \text{Rot}(y, \theta) \times \text{Rot}(z, \phi)]$$

→ by simply multiplying rotations about the x-axis through ψ , the y-axis through θ and the z-axis through ϕ , we can get the rotation from frame A \rightarrow D.

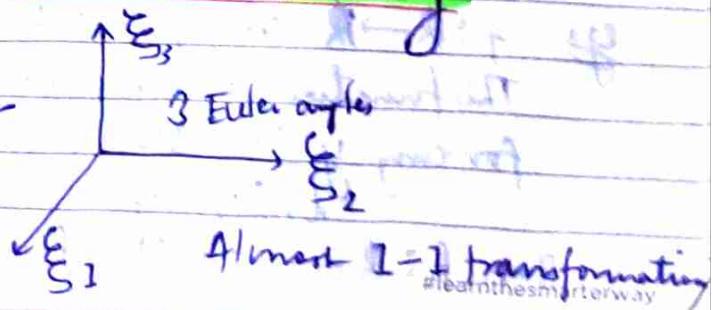
$$[{}^A R_D = \text{Rot}(x, \psi) \times \text{Rot}(y, \theta) \times \text{Rot}(z, \phi)]$$

roll pitch yaw

→ A 3D Euler, any rotation can be described by three successive rotations about linearly independent axes.

we can also use a 3x3 rotation matrix from the 3 Euler angles

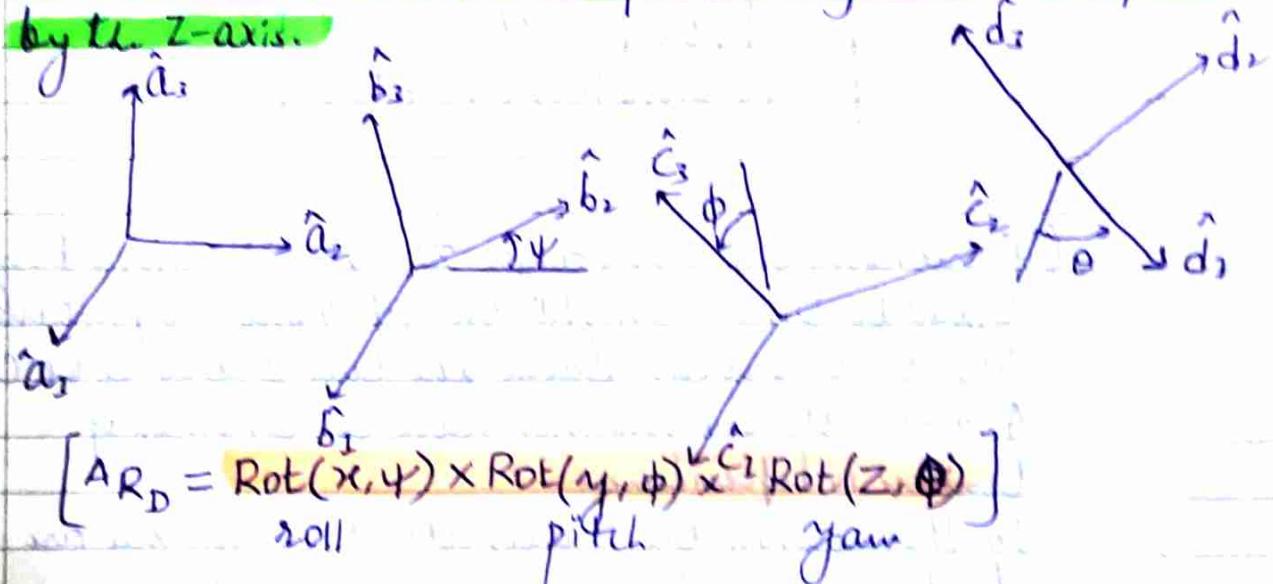
$$[3 \times 3 \text{ rotation matrix}]$$



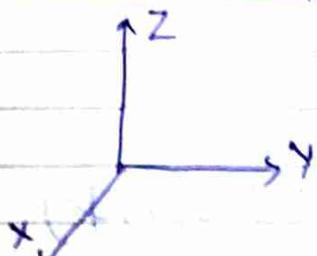
Is the reverse true? In other words, for every rotation matrix, is there a unique set of Euler angles?

In other words, does the map between the \mathbf{G} coordinates, the 3 Euler angles and the rotation matrix, is this map 1-1?

Ans - NO. It's almost one-to-one, but there are certain points that are analogous to the North Pole and the South Pole on the Earth's surface, at which the Euler angles are not well defined. This set of Euler angles is often called the **X-Y-Z Euler angles**, because the sequence in which rotation matrices are multiplied, relates to a rotation about the **X-axis**, followed by the **Y-axis**, followed by the **Z-axis**.



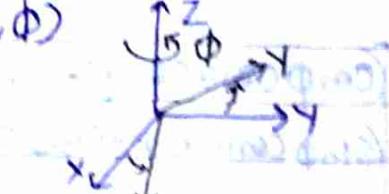
We can also have other types of Euler angles. This particular one is called a **Z-Y-Z Euler angles**.



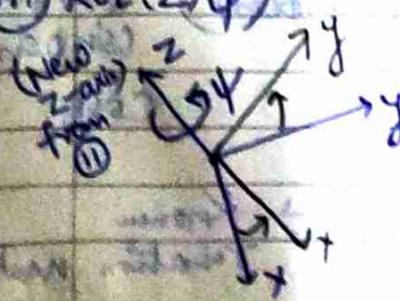
$$R = \text{Rot}(Z, \phi) \times \text{Rot}(Y, \theta) \times \text{Rot}(Z, \psi)$$

Sequence of three rotations about body-fixed axes -

① Rot(Z, φ)



③ Rot(Z, ψ)



② Rot(Y, θ)



- $\text{Rot}(z, \phi)$
 - $\text{Rot}(y, \theta)$
 - $\text{Rot}(z, \psi)$
- Are these linearly independent?
 (Although we have two rotations about z-axis, the question we have to ask is are they the same z-axis or different?)

→ we can see that the two z-axes are not collinear
 they are independent.
 → If this condition is satisfied, then the 3 angles are Euler angles and they can parameterize the set of rotations, i.e.,

- Three Euler angles
- ϕ, θ, ψ
- parameterize rotations

Note - $\theta = 0$ is a special (singular) case

(if we have an Euler angle zero, in this case, θ being equal to 0, we might ~~end up~~ run into problems).
 So, $\theta = 0$ would cause the two z-axes to be collinear. As a result, we will not have the condition of linear independence.

In other words, the 3 axes about which we are performing rotations are no longer independent.
 So, $\theta = 0$ is analogous to be being at the North or the South pole on the Earth surface.

Determination of Euler angles

$$R = \text{Rot}(z, \phi) \times \text{Rot}(y, \theta) \times \text{Rot}(z, \psi)$$

$$R = \begin{bmatrix} \cos\phi \cos\theta \cos\psi - \sin\phi \sin\psi & -\cos\phi \cos\theta \sin\psi - \sin\phi \cos\psi & \cos\phi \sin\theta \\ \sin\phi \cos\theta \cos\psi + \cos\phi \sin\psi & -\sin\phi \cos\theta \sin\psi + \cos\phi \cos\psi & \sin\phi \sin\theta \\ -\sin\theta \cos\psi & \sin\theta \sin\psi & \cos\theta \end{bmatrix}$$

$$= \begin{bmatrix} R_{11} & R_{12} & R_{13} \\ R_{21} & R_{22} & R_{23} \\ R_{31} & R_{32} & R_{33} \end{bmatrix}$$

→ Known
 + Rotation matrix

$$R_{33} = \cos \theta \rightarrow \theta = ?$$

$$R_{31} = -\sin \theta \cos \psi \quad]$$

$$R_{32} = \sin \theta \sin \psi \quad]$$

$$R_{13} = \cos \phi \sin \theta \quad]$$

$$R_{23} = \sin \phi \sin \theta \quad]$$

Any one can be used to

get ψ value, given θ is known from above

$\phi = ?$, given θ (from above)

(Any one can be used)

$$\text{If } |R_{33}| < 1, \quad \theta = G \arccos(R_{33}), \quad G = \pm 1$$

(we have two pieces of information for both ψ & ϕ because of that we don't use the standard inverse tangent function) \rightarrow we use something called the atan2 function. The atan2 function helps us overcome the ambiguity that exists in inverse tangent function

$$\text{If } R_{33} = 1,$$

$$\psi = \text{atan2}\left(\frac{R_{32}}{\sin \theta}, -\frac{R_{31}}{\sin \theta}\right)$$

$$\phi = \text{atan2}\left(\frac{R_{23}}{\sin \theta}, \frac{R_{13}}{\sin \theta}\right)$$

Twoblets
of Euler
angles for
every R
for almost
all R_{33} !

$$R = \begin{bmatrix} \cos \phi \cos \psi - \sin \phi \sin \psi & -\cos \phi \sin \psi - \sin \phi \cos \psi & 0 \\ \cos \phi \sin \psi + \sin \phi \cos \psi & -\sin \phi \sin \psi + \cos \phi \cos \psi & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

$$\text{If } R_{33} = -1,$$

Infinite set of Euler angles

$$R = \begin{bmatrix} -\cos \phi \cos \psi - \sin \phi \sin \psi & \cos \phi \sin \psi - \sin \phi \cos \psi & 0 \\ \cos \phi \sin \psi - \sin \phi \cos \psi & \sin \phi \sin \psi + \cos \phi \cos \psi & 0 \\ 0 & 0 & -1 \end{bmatrix}$$

$f(\phi + \psi) \rightarrow$ If $R_{33} = \pm 1$, then in that case θ is going to be equal to either 0 or π and we have two alternatives.

\rightarrow In both these cases, we will see from the grouping of terms that the rotation matrices are functions only of the sum of two angles, ψ and ϕ .

In other words, it's impossible to determine either ψ or ϕ uniquely.

The tangent function

$$\tan \theta = \frac{\sin \theta}{\cos \theta} = \frac{y}{x}$$

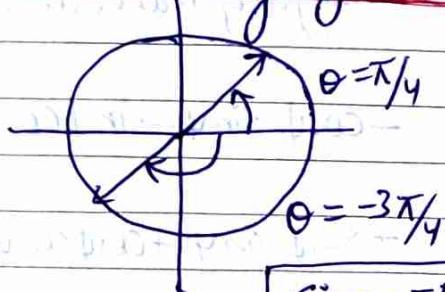
$\theta = \tan^{-1}(y/x)$ return the angles for which $\tan \theta = y/x$

→ The inverse tangent function is implemented in programming languages as the 'atan' function.

$$\text{atan}(y/x) = \tan^{-1}(y/x)$$

→ Shortcomings of This function -

①



$$\sin \theta = \frac{\sqrt{2}}{2}$$

$$\cos \theta = -\frac{\sqrt{2}}{2}$$

$$\tan \theta = 1$$

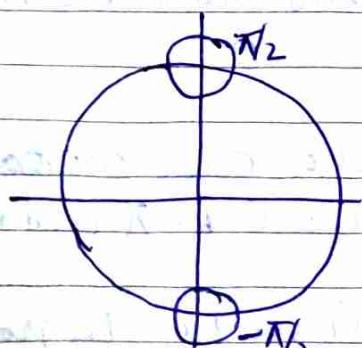
$$\begin{aligned} \sin \theta &= \frac{\sqrt{2}}{2} \\ \cos \theta &= -\frac{\sqrt{2}}{2} \\ \tan \theta &= 1 \end{aligned}$$

$$\begin{aligned} \tan^{-1}\left(\frac{1}{\sqrt{3}}\right) &= \frac{\pi}{6} \\ \tan^{-1}\left(-\frac{1}{\sqrt{3}}\right) &= -\frac{\pi}{6} \end{aligned}$$

The atan function cannot distinguish between these two angles.

In fact, the atan function cannot distinguish between any pair of points opposite from each other on the unit circle.

②



→ A second problem with the atan function occurs at the angles $-\pi/2$ and $\pi/2$.

$$\cos(-\pi/2) = \cos(\pi/2) = 0$$

$$\frac{\sin \theta}{\cos \theta} = \frac{y}{x} = \frac{\pm 1}{0} = \text{undefined}$$

atan's function fails at these points. Returns values in range $(-\pi/2, \pi/2)$ #earlier manner

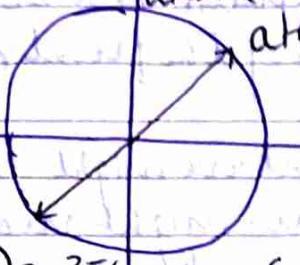
To avoid these problems, the atan function can only return angles in the range $(-\pi/2, \pi/2)$ (non-inclusive)

atan2 function

→ $\text{atan2}(y, x)$ is an implementation of the atan function that takes into account value and signs of y and x .

$$\text{atan2}(1, 0) = \pi/2$$

$$\text{atan2}(1, 1) = \pi/4$$



Returns values in range $[0, 2\pi)$

$$\text{atan2}(-1, 1) = -3\pi/4 \quad \text{atan2}(-1, 0) = -\pi/2$$

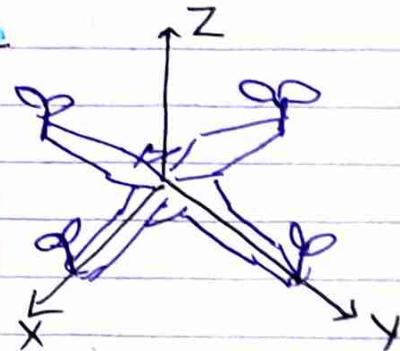
$X \longrightarrow X$

- Given one we can determine the other but it's impossible to disambiguate between ϕ and ψ because the groupings only contain terms that combine them.
- So, when $R_{33} = \pm 1$, we have infinite set of Euler angles.

Note - In most of our work, we use a different set of Euler angles.

Z-X-Y Euler Angles

- ~~Order~~ so-called because the 3 rotation matrices we consider are the rotation about the Z -axis through ψ , followed by the rotation about the X -axis through ϕ and then the rotation of the y -axis through θ , i.e.,



Sequence of the rotations about body-fixed axes

- Rot (Z, ψ)
- Rot (X, ϕ)
- Rot (Y, θ)

Verify (by multiplying the rotations, we get a rotation matrix of below form)

$$R = \begin{bmatrix} c\psi c\theta - s\phi s\psi s\phi & -c\phi s\psi & c\psi s\theta + c\theta s\phi s\psi \\ c\psi s\theta - c\theta s\phi s\phi & c\phi c\psi & s\psi s\theta - c\theta s\phi c\psi \\ -c\phi s\theta & s\phi & c\phi c\theta \end{bmatrix}$$

- for every set of Euler angles, we might have at least two solutions for Euler angles for a given rotation matrix. At some points, we can have infinite solutions.
- what we really want is a second set of Euler angles to take care of the points at which we have ∞ solutions.

So, this suggests that we might have many sets of Euler angles that we might want to consider so that we don't have a point at which we have ∞ solutions.

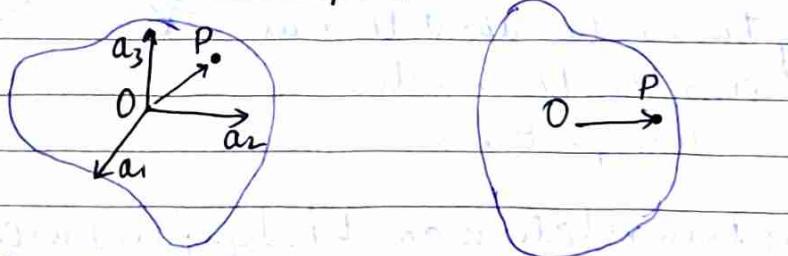
Q: What is the minimum number of set of Euler angles to cover all of the rotation group $SO(3)$?

$$SO(3) = \{R \in \mathbb{R}^{3 \times 3} \mid R^T R = R R^T = I\}$$

Euler's Theorem

Rotations - Any displacement of a rigid body such that a point on the rigid body, say O , remains fixed, is equivalent to a rotation about a fixed axis through the point O .

Rotation with O fixed



Now, translate the rigid body from second position & orientation so that the two origins are identical so that point P moves to new point P' .

$$\begin{aligned} \overrightarrow{OP} &= p_1 \hat{a}_1 + p_2 \hat{a}_2 + p_3 \hat{a}_3 \\ \overrightarrow{OP'} &= q_1 \hat{a}'_1 + q_2 \hat{a}'_2 + q_3 \hat{a}'_3 \\ \vec{q} &= \vec{R} \vec{p} \end{aligned}$$

$$\begin{bmatrix} q_1 \\ q_2 \\ q_3 \end{bmatrix} = \begin{bmatrix} R_{11} & R_{12} & R_{13} \\ R_{21} & R_{22} & R_{23} \\ R_{31} & R_{32} & R_{33} \end{bmatrix} \begin{bmatrix} p_1 \\ p_2 \\ p_3 \end{bmatrix}$$

#Learn the smarter way

Q Is there ~~any~~ an axis that remains fixed under rotation?

In other words, is there a point P such that after the rotation through R , it stays fixed, i.e., Point P is invariant to the rotation?

Proof of Euler's Theorem

$$\vec{q} = R \vec{p}$$

Is there a point \vec{p} that maps onto itself?

→ If there were such a point p , then $\vec{p} = R \vec{p}$

(this is nothing but a

Solve eigenvalue problem

$$R \vec{p} = \lambda \vec{p}$$

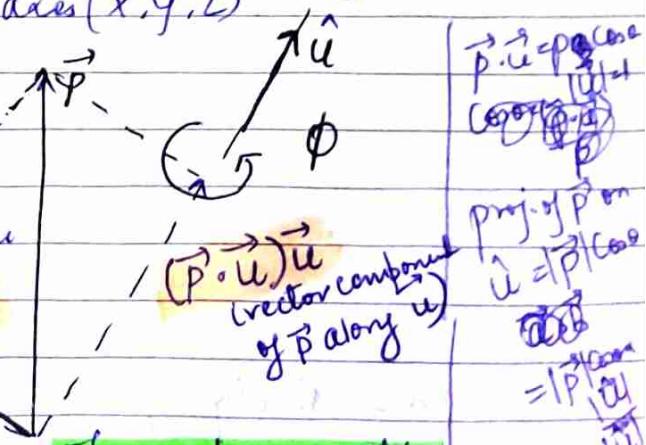
(verify $\lambda=1$ is an eigenvalue
for any R)

statement that 1 is the eigenvalue of the rotation matrix R)

How does one find the rotation matrix for a general axis and angle of rotation?

Note - we already know the answer if the axis of rotation is one of the coordinate axes ($\hat{x}, \hat{y}, \hat{z}$)

Suppose we have an arbitrarily oriented axis given by the unit vector \hat{u} and an arbitrary angle of rotation, ϕ . Let's consider a generic vector \vec{p}



Let's see how the vector \vec{p} gets rotated if the rigid body to which it's attached gets rotated about the axis \hat{u} through an angle ϕ

The component of the vector that's along the axis is never changed by the rotation.

→ In other words, the vector

$(\vec{p} \cdot \hat{u}) \hat{u}$ remains unchanged by the rotation R .

→ But the L component which is along \hat{v} will get rotated

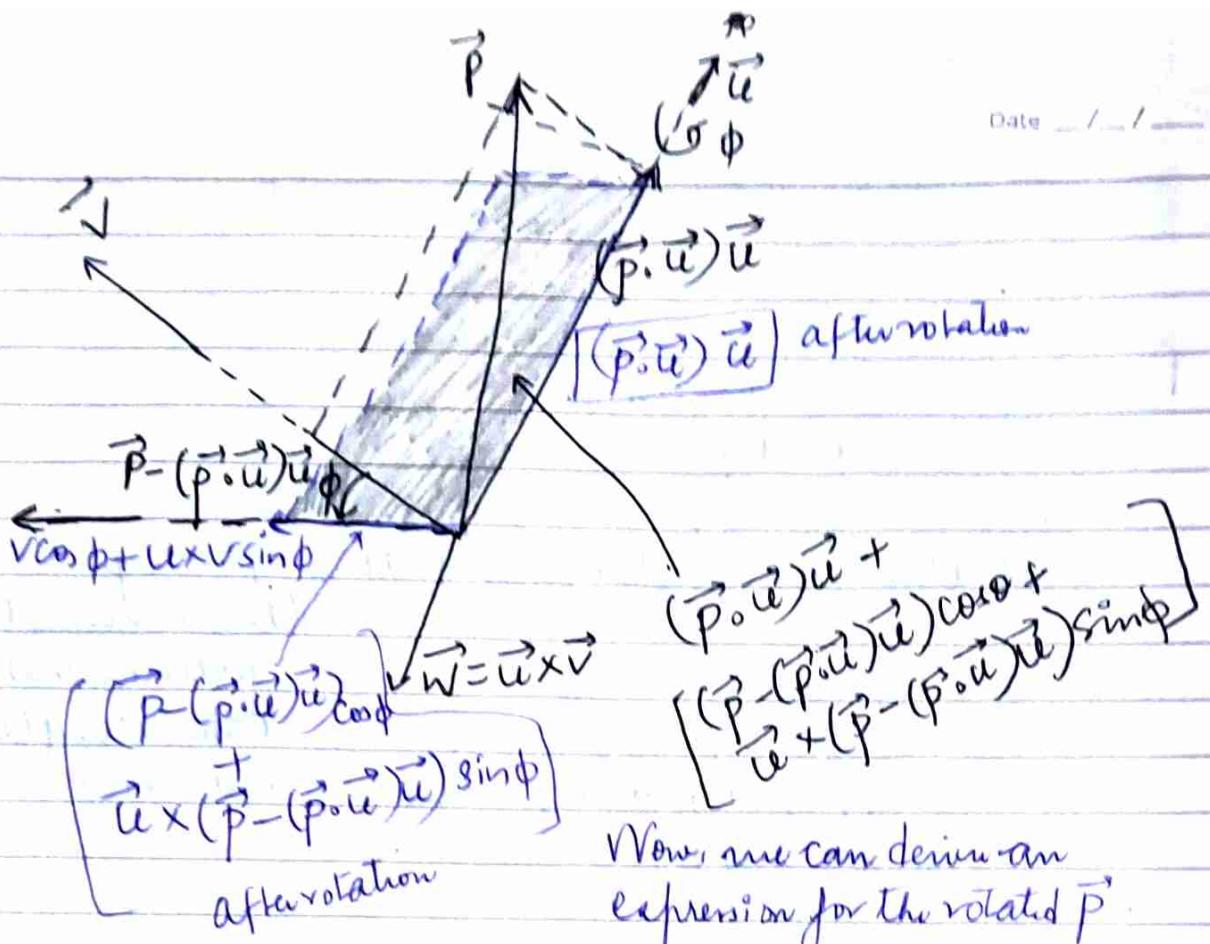
let's fix \hat{u}, \hat{v} & orthogonal vector \hat{w} ,

where

$$\hat{w} = \hat{u} \times \hat{v}$$

; as

local coordinate system or local basis



Now, we can derive an expression for the rotated \vec{P}

$$\vec{P} \cos \phi + \vec{u} \vec{u}^T (1 - \cos \phi) \vec{P} + \vec{u} \times \vec{p} \sin \phi$$

(this expression holds for any vector of \vec{P} since it's a generic vector)

Let $\vec{u} = \vec{u} \vec{u}^T$

↳ Skew-symmetric matrix whose components are the

Name as components of \vec{u}
 matrix \Rightarrow { Symmetric, $A^T = A$ }
 Skew-symmetric,
 $A^T = -A$

multiple zeros on the
diagonal

General expression for 3×3
Skew-symmetric matrix

$$A = \begin{bmatrix} 0 & -A_{21} & A_{13} \\ A_{21} & 0 & -A_{32} \\ A_{13} & A_{32} & 0 \end{bmatrix}$$

$$\vec{u} \downarrow$$

$$\vec{a} = \begin{bmatrix} A_{32} \\ A_{13} \\ A_{21} \end{bmatrix} = \begin{bmatrix} a_1 \\ a_2 \\ a_3 \end{bmatrix}$$

$$A = \begin{bmatrix} 0 & -A_{21} & A_{13} \\ A_{21} & 0 & -A_{32} \\ A_{13} & A_{32} & 0 \end{bmatrix}$$

(only has 3 independent parameters)

we use the hat operator to switch between these two ~~vector~~ representations

Date: 2023/10/1

$$\hat{\vec{a}} = \begin{bmatrix} \vec{a}^* \\ \vec{a}_1 \\ \vec{a}_2 \\ \vec{a}_3 \end{bmatrix} = \begin{bmatrix} 0 & -a_3 & a_2 \\ a_3 & 0 & -a_1 \\ -a_2 & a_1 & 0 \end{bmatrix}$$

Vector cross Product — The hat operator is also used to denote the cross product between two vectors -

$$\vec{u} \times \vec{v} = \hat{u} \vec{v}$$

$$\vec{u} \times \vec{v} = \begin{bmatrix} 0 & -u_3 & u_2 \\ u_3 & 0 & -u_1 \\ -u_2 & u_1 & 0 \end{bmatrix} \begin{bmatrix} v_1 \\ v_2 \\ v_3 \end{bmatrix}$$

↑
[u x]

Representation of Angular Velocities

angular velocity vector: $\hat{\vec{\omega}}^b = \vec{R}^T \dot{\vec{R}}$

$$\hat{\vec{\omega}}^s = \dot{\vec{R}} \vec{R}^T$$

$\vec{R}^T \dot{\vec{R}}$ and $\dot{\vec{R}} \vec{R}^T$ are skew-symmetric

Axis/Angle to Rotation Matrix

Rotation of a generic vector \vec{p} about \vec{u} through ϕ

$$R_p = p \cos \phi + \vec{u} \vec{u}^T (1 - \cos \phi) p + \vec{u} \vec{u} \sin \phi$$

Rodrigues formula

$$\text{Rot}(e, \phi) = I \cos \phi + \vec{u} \vec{u}^T (1 - \cos \phi) + \vec{u} \vec{u} \sin \phi$$

Axis of rotation \vec{u}
Rotation angle ϕ

So, given a generic vector \vec{u} and a rotation angle ϕ , we can define the rotation matrix using this closed form expression.

- ④ Set \vec{u} to be a unit vector along x (or y or z). Verify result is the same as $\text{Rot}(x, \phi)$.

② Is the ~~(axis, angle)~~ to rotation matrix map onto? $1 \rightarrow 1$?

\swarrow
 $\text{Rot}(\mathbf{u}, \phi)$ and $\text{Rot}(-\mathbf{u}, 2\pi - \phi)$?
 restrict ϕ to the interval $[0, \pi]$?

* for a given rotation matrix, can we extract \mathbf{u} and ϕ from this matrix?

Let's extract this axis and the angle from the rotation matrix, R .

Verify, $\cos \phi = \frac{\text{Tr} - 1}{2}$, Tr is the trace of the rotation matrix.

$$\text{Tr} = (R_{11} + R_{22} + R_{33})$$

If above is true, then ~~we~~.

from Rodrigues's formula,

$$\hat{\mathbf{u}} = \frac{1}{2 \sin \phi} (R - R^T) \quad (\text{we get } \mathbf{u}, \text{ without solving for eigenvector})$$

→ (axis, angle) to rotation matrix map is many to 1.

→ restricting angle to the interval $[0, \pi]$ makes it $1 \rightarrow 1$ except for

$$\text{Tr} = 3 \Rightarrow \phi = 0 \Rightarrow \text{no unique axis of rotation}$$

$$\text{Tr} = -1 \Rightarrow \phi = \pi \Rightarrow \text{two axes of rotation}$$

(either the axis we obtained using the formula above ~~or~~ or the opposite)
 i.e., \mathbf{u} or $-\mathbf{u}$

* The best way to think about the rotation group, $\text{SO}(3)$ is by imagining a solid ball of radius $\pi/2$.

Any point on the ball's surface or inside the ball

essentially represents a rotation whose axis

is given by the radius and the vector that describes

the point with the origin of sphere as the origin of the vector ~~falls outside~~ ~~axis of rotation~~

$\text{SO}(3)$

axis of rotation

Again, it seems like there is 1-1 map between points on the surface of the ball or inside the ball & the set of rotations. This is true, except for points that are on the surface of the ball.

So, if we look at any pair of points on the ball, but they are diametrically opposite, They essentially correspond to the same rotation.

Rotations and angular Velocities

Rate derivatives of rotations

Rotation matrix, $R(t)$

Orthogonality

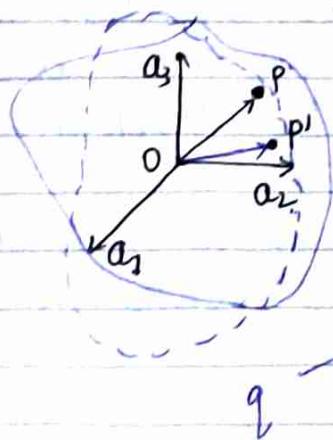
$$R^T(t)R(t) = I \xrightarrow{\frac{d}{dt}(0)} \dot{R}^T R + R^T \dot{R} = 0$$

$$R(t)R^T(t) = I \xrightarrow{\dot{R}^T + R R^T = 0}$$

$\dot{R}^T R$ and $\dot{R} R^T$ are skew symmetric.

(In other words, there ~~are 6~~ are only 3 independent elements in these skewsymmetric matrices).

Rotation with O fixed



$$\vec{OP} = p_1 \hat{a}_1 + p_2 \hat{a}_2 + p_3 \hat{a}_3$$

$$\vec{OP}' = q_1 \hat{a}_1 + q_2 \hat{a}_2 + q_3 \hat{a}_3$$

$$\begin{bmatrix} q_1 \\ q_2 \\ q_3 \end{bmatrix} = \begin{bmatrix} R_{11} & R_{12} & R_{13} \\ R_{21} & R_{22} & R_{23} \\ R_{31} & R_{32} & R_{33} \end{bmatrix} \begin{bmatrix} p_1 \\ p_2 \\ p_3 \end{bmatrix}$$

$$q(t) = R(t)p$$

Constant
coordinates of
P in body fixed
frame

Changing coordinates of P
as the rigid body rotates

diff. both sides, we get

$$\dot{q} = \dot{R}P_R \quad (P = \text{const.})$$

(velocity in
inertial frame)

position in
body-fixed frame

$$R^T \dot{q} = R^T \dot{R} p \quad (\text{premultiply both sides by the transpose of the rotation matrix})$$

Date: _____

velocity in body fixed frame encodes angular velocity in body-fixed frame

$$\dot{q} = \underline{R R^T} \dot{q}$$

$\overset{\circ}{\text{W}} \overset{\circ}{\text{S}}$

not in body frame,
but spatial
angular velocity

(velocity
in inertial
frame) encodes angular
velocity in
inertial frame

→ we have seen that skew-symmetric matrices encode cross products. So, about this equations is essentially our ability to generate velocities by taking the cross product of an anylen velocity vector with a position vector.

① In the first eqn. it's the angular velocity in the body-fixed frame which yields the velocity in the body-fixed frame.

(+) In the second expr, it's the angular velocity in the inertial frame which then yields the velocity in the inertia frame.

The first one is written in terms of basis vectors on the body-fixed frame, and the second one is written in terms of basis vectors in inertial frame.

8) What's the angular velocity for a rotation about z-axis?

$$R = \begin{bmatrix} \cos \theta & -\sin \theta & 0 \\ \sin \theta & \cos \theta & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

$$R^T = \begin{bmatrix} \cos \theta & \sin \theta & 0 \\ -\sin \theta & \cos \theta & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

$$R = \begin{bmatrix} \cos \theta & -\sin \theta & 0 \\ \sin \theta & \cos \theta & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

Angular velocity for a rotation about the Z-axis.

$$R = \begin{bmatrix} \cos\theta & -\sin\theta & 0 \\ \sin\theta & \cos\theta & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

Date: _____

$$R^T \dot{R} = \begin{bmatrix} \cos\theta & \sin\theta & 0 \\ -\sin\theta & \cos\theta & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} -\sin\theta & -\cos\theta & 0 \\ \cos\theta & -\sin\theta & 0 \\ 0 & 0 & 0 \end{bmatrix} \dot{\theta}$$

$$\begin{aligned} \dot{R}^T R = \dot{\theta} & \begin{bmatrix} -\sin\theta & -\cos\theta & 0 \\ \cos\theta & -\sin\theta & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} \cos\theta & \sin\theta & 0 \\ -\sin\theta & \cos\theta & 0 \\ 0 & 0 & 1 \end{bmatrix} \\ & = \begin{bmatrix} 0 & -1 & 0 \\ 1 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix} \dot{\theta} = \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} \dot{\theta} \end{aligned}$$

$R^T \dot{R} = \dot{R}^T R$
This happens in this special case when ω is constant.

In this particular case, this symmetric matrix corresponds to the $(0, 0, 1)$ vector which is the Z-axis. If we rotate a rigid body about the Z-axis and the axis of rotation is constant, then the angular velocity vector will also be along the Z-axis.

Two Rotations

What happens if our rotation is obtained by composing two rotations?

$$R = R_z(\theta) R_x(\phi)$$

• In this case the rotations are about the Z-axis through θ , about the X-axis through ϕ)

$$\dot{\omega}_b^b = R^T \dot{R} = (R_z R_x)^T (R_z \dot{R}_x + R_z R_x \dot{R}_x)$$

$$= R_x^T R_z^T \dot{R}_z R_x + R_x^T \dot{R}_x$$

depends only on the rate of change of R_z depends only on the rate of change of R_x

$$\begin{aligned}
 \dot{W}^s &= \dot{R}R^T = (\dot{R}_Z R_X + R_Z \dot{R}_X)(R_Z R_X)^T \\
 &= \underline{\dot{R}_Z R_Z^T} + \underline{R_Z \dot{R}_X R_X^T R_Z^T} \\
 &\quad \text{depends on} \quad \text{depends on the} \\
 &\quad \text{the value of} \quad \text{rate of change of} \\
 &\quad \text{charge of } R_Z
 \end{aligned}$$

Date: ___/___/___

Spatial
angular
Velocity

Above two expression are different. In both cases, they consist of two terms. One depends on θ and the second depends on ϕ but because the axis of rotation is not fixed, the two expressions are different. The body fixed angular velocity and the spatial angular velocity have different expressions.

Dynamics of a Quadrotor

two coordinate systems, one attached to the moving robot & the other, the inertial coordinate system

~~Outer Angles~~
 $Z \times^q$ convolution
uses $N/1$

Note

Z-X-4 Euler Angles

$$-\text{Rot}(z, \psi)$$

$$- \text{Rot}(x, \phi)$$

- Rot(y, θ)

→ Singularity occurs when the roll angle is equal to 0 (i.e. $\phi = 0$)

Even when $\phi \neq 0$, we can have two sets of Euler angles for every rotation.

$$F_i = k_F \omega_i^2$$

$$M_i = k_M \omega_i^2$$

Relevant forces & moments

~~External forces & moments~~

$$\vec{F} = \vec{F}_1 + \vec{F}_2 + \vec{F}_3 + \vec{F}_4 - mg \vec{a}_3$$

$$\vec{M} = \vec{a}_1 \times \vec{F}_1 + \vec{a}_2 \times \vec{F}_2 + \vec{a}_3 \times \vec{F}_3 + \vec{a}_4 \times \vec{F}_4 +$$

$$\vec{M}_1 + \vec{M}_2 + \vec{M}_3 + \vec{M}_4$$

reaction moments

moment of thrust from F_4

→ To predict the net acceleration, we have to write down the equation of motion. These come from Newton & Euler and they are called the Newton-Euler Equation.

Newton-Euler Equation

Newton's law of motion for a Single Particle of mass m

Gravity: $\vec{F} = m\vec{a}$ (Newton's Second Law)

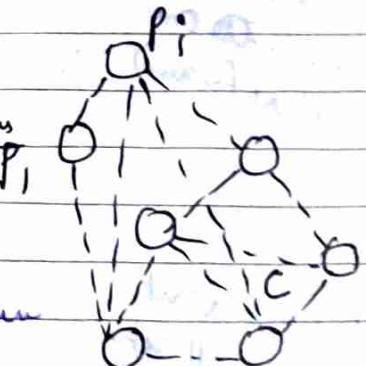
for a single particle

Newton's Second Law for a System of particles

Centre of mass →

$$\vec{r}_c = \frac{1}{m} \sum_{i=1}^N m_i \vec{r}_i$$

$i=1, N$ (Weighted sum of all the position vectors)



→ The center of mass for a system of particles, S , accelerates in an inertial frame (A) as if it were a single particle with mass m (equal to the total mass of the system) acted upon by a force equal to the net external force.

P_A

$$\vec{F} = \sum_{i=1}^N \vec{F}_i = m \frac{d\vec{r}_c}{dt}$$

The Superscript A refers to the fact that we are computing all these quantities in an inertial frame A.

The Superscript C refers to the fact that we are computing the rate of change of linear momentum.

Superscript A refers to the fact that we are computing all these quantities in an inertial frame A.

Law of Change of Linear momentum

Date: ___/___/___

Derivation $\vec{F} = \frac{d\vec{P}}{dt}$, Linear momentum of the system of particles in the inertial frame A.

frame. This (in order for this equation to be valid)

(Also true for a rigid body)

nothing but a set of infinite particles, all glued together rigidly.

So, if above eqn is valid for a set of particles, it must be valid for a collection of particles.

What's the equivalent for rotational motion?

Rotational equations of motion for a rigid body.

The rate of change of angular momentum of the rigid body B relative to C in A is equal to the resultant moment of all external forces acting on the body relative to C.

* differentiation must be done in an inertial frame

$$\frac{d \vec{A} \vec{H}_C^B}{dt} = \vec{M}_C^B$$

Net moment applied to the rigid body with the origin C, the center of mass, in the inertial frame A.

Net moment from all ext. forces & torques about reference C

(Take all the ext. forces & compute their moments & add these moments to all ext. couples or torques)

→ Here also the truly subscript C tells us that we are computing with C as an origin

$$\vec{A} \vec{H}_C^B = \vec{I}_C \cdot \vec{\omega}$$

(Then computation has to be done in 3D)

$\vec{I} \rightarrow$ a 3D vector
 $\vec{\omega} \rightarrow$ a 3D vector
with C as the origin

$\vec{A} \vec{L}^B \rightarrow$ also obtained in the inertial frame

(Angular velocity of B in A)

Principal Axes and Principal Moments

Date: ___/___/___

Principal Axis of inertia

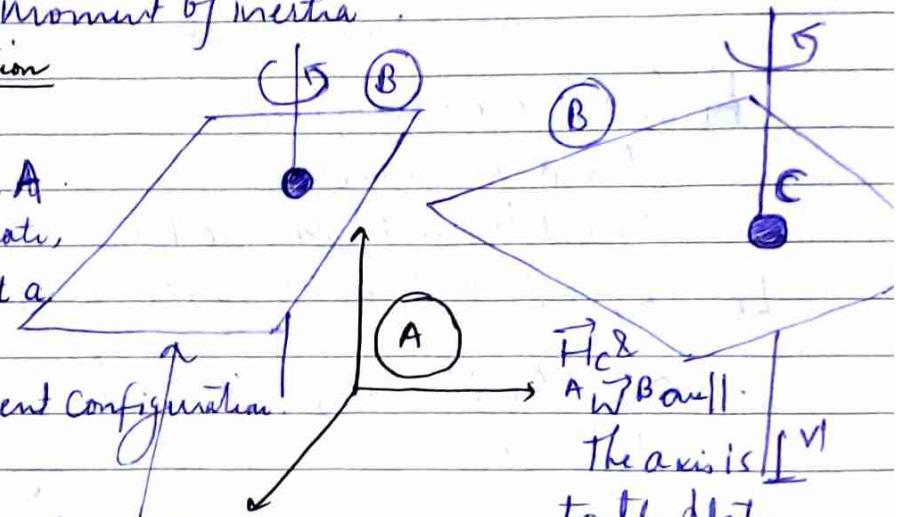
\hat{u} is a unit vector along a principal axis if $\vec{I} \cdot \hat{u}$ is parallel to \hat{u} . There are 3 independent principal axes (In other words, there are 3 independent axes such that \vec{I} times the unit vector along that axis will give us a vector that's parallel to that axis). The moment of inertia is essentially the scaling term.

Principal moment of inertia

The moment of inertia w.r.t. a principal axis, $\vec{u} \cdot \vec{I} \cdot \vec{u}$, is called a principal moment of inertia.

Physical Interpretation

We have a rigid frame, A.
We have a parallel plate,
that's spinning about a
vertical axis, but
there are two different configurations



the configuration is
symmetric but the
axis is not \perp to the plate.

The axis is \perp to the plate
In fact this configuration
is symmetric.

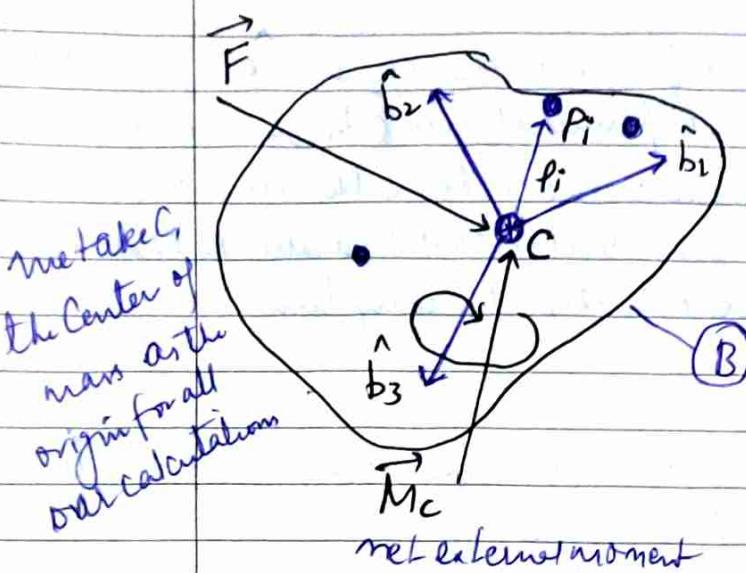
\rightarrow Angular momentum
vector & angular
velocity vector are
not parallel, because
the axis of rotation
does not coincide with
any of the principal axes.

If we compute
the angular momentum
we will find that
the angular momentum
vector & angular
velocity vector are
parallel.

Euler's Equation (tells us the rotational equation of motion)

Date: / /

$$\frac{d\vec{H}_C}{dt} = \vec{M}_C \quad (1)$$



The rate of change of angular momentum is equal to the net moment applied to the rigid body.

$\vec{b}_1, \vec{b}_2, \vec{b}_3$ are a set of body fixed unit vectors that define a body fixed frame.

Let $\vec{b}_1, \vec{b}_2, \vec{b}_3$ be along the principal axes

$$\text{and } {}^A\vec{w}^B = \omega_1 \vec{b}_1 + \omega_2 \vec{b}_2 + \omega_3 \vec{b}_3$$

diff. (1),

$$\frac{d\vec{H}_C}{dt} + {}^A\vec{w}^B \times \vec{H}_C = \vec{M}_C$$

(involves the derivative in a body fixed frame)

(Correction factor, takes into account

the fact that the inertia matrix times the angular velocity vector differentiation is

done in the body fixed frame)

Correction factor is

simply the angular velocity of the moving body fixed frame crossed with the angular momentum.

→ (1) Above correction factor is a well-known fact in mechanics.

Anytime we differentiate a vector in a moving frame, its derivative is different from the derivative in a fixed frame.

* first term can be written in terms of inertia matrix and the angular velocity vector. differentiation is done in the body fixed frame

Because we have chosen principal axes, it turns out that the off diagonal elements in the inertia tensor are zero.

* the difference is obtained by multiplying the cross product of angular velocity with that vector

$$\frac{d\vec{H}_c}{dt} = I_{11} \vec{w}_1 \vec{b}_1 + I_{22} \vec{w}_2 \vec{b}_2 + I_{33} \vec{w}_3 \vec{b}_3$$

$(\vec{A} \vec{W}^B \times \vec{H}_c)$ can be written in the component form as -

Second term

$$\begin{bmatrix} I_{11} & 0 & 0 \\ 0 & I_{22} & 0 \\ 0 & 0 & I_{33} \end{bmatrix} \begin{bmatrix} \vec{w}_1 \\ \vec{w}_2 \\ \vec{w}_3 \end{bmatrix} + \begin{bmatrix} 0 & -w_3 & w_2 \\ w_3 & 0 & -w_1 \\ -w_2 & w_1 & 0 \end{bmatrix} \begin{bmatrix} I_{11} & 0 & 0 \\ 0 & I_{22} & 0 \\ 0 & 0 & I_{33} \end{bmatrix} \begin{bmatrix} \vec{w}_1 \\ \vec{w}_2 \\ \vec{w}_3 \end{bmatrix}$$

(Euler's equation of motion)

derivation of angular momentum in a body fixed frame

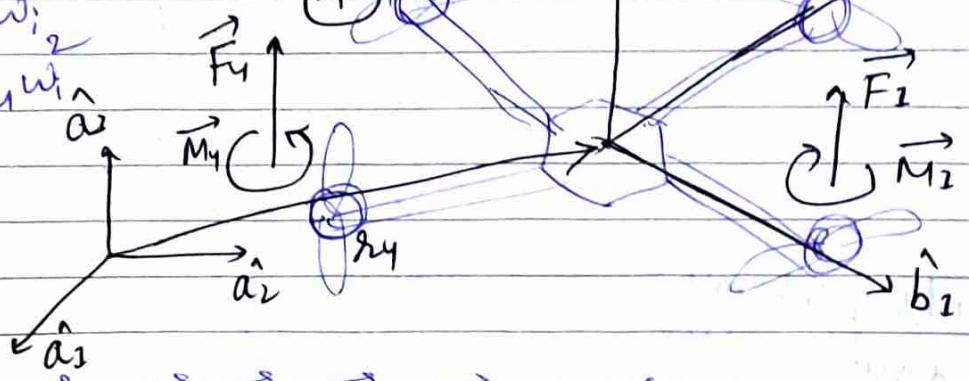
Net moment

(Correction term)

Quadrrotor Equations of Motion

$$F_i = k_F \vec{w}_i^2$$

$$M_i = k_M \vec{w}_i$$



$$\text{Net force } \vec{F} = \vec{F}_1 + \vec{F}_2 + \vec{F}_3 + \vec{F}_4 - mg \vec{a}_3$$

$$\text{Net moment } \vec{M} = \vec{a}_1 \times \vec{F}_1 + \dots + \vec{a}_4 \times \vec{F}_4 + \vec{M}_1 + \dots + \vec{M}_4$$

If we combine the net force and the net moment with the Newton-Euler Equations, we get these two sets of eqn

$$\text{Rot. of thrust vector } \vec{A} \vec{W}^B = \vec{p} \vec{b}_1 + \vec{q} \vec{b}_2 + \vec{r} \vec{b}_3$$

$$\vec{m}_x = \begin{bmatrix} 0 \\ 0 \\ -mg \end{bmatrix} + \vec{R} \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix}$$

$$I \begin{bmatrix} \dot{p} \\ \dot{q} \\ \dot{r} \end{bmatrix} = \begin{bmatrix} L(F_2 - F_4) \\ L(F_3 - F_1) \\ M_1 + M_2 + M_3 + M_4 \end{bmatrix} - \begin{bmatrix} p \\ q \\ r \end{bmatrix} \times I \begin{bmatrix} \dot{p} \\ \dot{q} \\ \dot{r} \end{bmatrix}$$

Net moment M_2 (in the body fixed frame) over

How do we estimate/calculate all the parameters in this model?

Date: _____

$$\ddot{\vec{m}\vec{r}} = \begin{bmatrix} 0 \\ 0 \\ mg \end{bmatrix} + R \begin{bmatrix} 0 \\ 0 \\ \vec{F}_1 + \vec{F}_2 + \vec{F}_3 + \vec{F}_4 \end{bmatrix}$$

$$\vec{I} \begin{bmatrix} \dot{p} \\ \dot{q} \\ \dot{r} \end{bmatrix} = \begin{bmatrix} L(\vec{F}_2 - \vec{F}_1) \\ L(\vec{F}_3 - \vec{F}_1) \\ M_1 - M_2 + M_3 - M_4 \end{bmatrix} - \begin{bmatrix} p \\ q \\ r \end{bmatrix} \vec{I} \begin{bmatrix} p \\ q \\ r \end{bmatrix}$$

Length

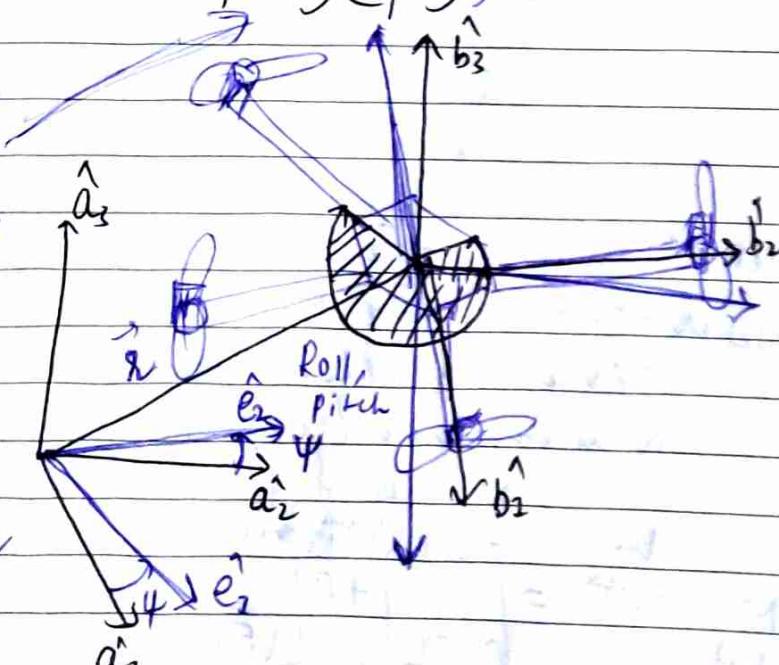
If we have a system that allowed to measure positions, velocities & accn, it's actually not too hard to estimate lengths, the masses & the inertia. It's also quite easy to calculate the angular velocity in the body fixed frame.

Angular velocity components in B

$$\begin{bmatrix} p \\ q \\ r \end{bmatrix} = \begin{bmatrix} \cos\theta & 0 & -\cos\phi\sin\theta \\ 0 & 1 & \sin\phi \\ \sin\theta & 0 & \cos\phi\cos\theta \end{bmatrix} \begin{bmatrix} \dot{\phi} \\ \dot{\theta} \\ \dot{\psi} \end{bmatrix}$$

Pitch
Roll
Yaw

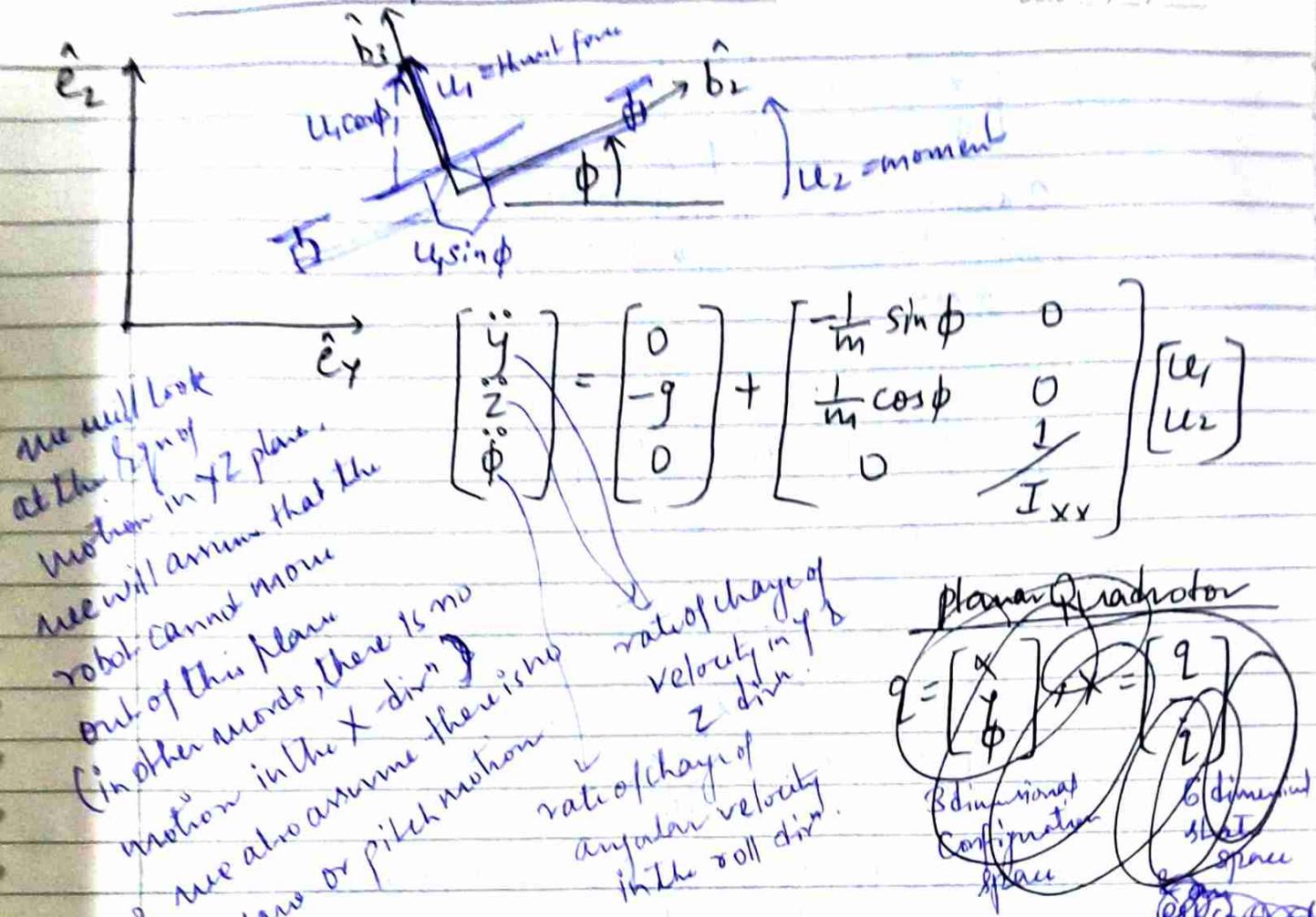
If we know the pitch, roll & yaw and we know the rate of change of the pitch, roll & yaw angles on the right side, a simple transformation yields the angular velocity components p, q & r along b_1, b_2 & b_3 .



This model involves 3 components of position, velocity & accn, three components of rotation, angular velocities & angular accelerations.

Planar Quadrotor Model

Date: / /



$$\begin{bmatrix} \ddot{y} \\ \ddot{z} \\ \ddot{\phi} \end{bmatrix} = \begin{bmatrix} 0 \\ -g \\ 0 \end{bmatrix} + \begin{bmatrix} -\frac{1}{m} \sin\phi & 0 \\ \frac{1}{m} \cos\phi & 0 \\ 0 & 1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \\ u_3 \end{bmatrix}$$



To describe the dynamics of system, it's useful to define a state vector

State Space for Quadrotors

State Vectors

- q describes the configuration (position) of the system
- x describes the state of the system

$$q = \begin{bmatrix} x \\ y \\ z \\ \phi \\ 0 \end{bmatrix}, \quad x = \begin{bmatrix} q_1 \\ q_2 \end{bmatrix}$$

state vector
includes the configuration & its derivative

Equilibrium Configuration

- q_e describes the equilibrium configuration of the system
- x_e describes the equilibrium state of the system

$$q_e = \begin{bmatrix} x_0 \\ y_0 \\ z_0 \\ 0 \\ 0 \end{bmatrix}, \quad x_e = \begin{bmatrix} q_{e1} \\ q_{e2} \\ 0 \end{bmatrix}$$

this gives us a 2D vector when both 6D state vectors are zero. A much smarter way

0 roll angle
0 pitch angle
any yaw angle

State vector for planar Quadrotor

Date: ___/___/___

$$q = \begin{bmatrix} \gamma \\ z \\ \phi \end{bmatrix}, x = \begin{bmatrix} \dot{q} \\ \ddot{q} \end{bmatrix}$$

3 dimensional
Configuration
space

a six dimensional
state space

Equilibrium State/Configuration

$$q_e = \begin{bmatrix} \gamma_0 \\ z_0 \\ 0 \end{bmatrix}, x_e = \begin{bmatrix} \dot{q}_e \\ \ddot{q}_e \\ 0 \end{bmatrix}$$

Dynamical Systems in State-Space form

↳ System where the
effects of action do not
occur immediately

- Evolution of the system's states is governed by a set of ordinary differential equations
- Ordinary differential equations are often rearranged into state-space form.

$$\dot{x} = f(x, u)$$

↑ ↑
matrices

x is a matrix of states
 u is a matrix of inputs

State-space form

Given an DDE:

- ① Identify the order, n , of the system
- ② Define the states

$$x_1 = y(t), x_2 = \dot{y}(t), \dots, x_n = y^{(n-1)}(t)$$

$(n-1)^{\text{th}}$ derivative

- ③ Create the state vector $x = [x_1 \ x_2 \ \dots \ x_n]^T$

$$= [y \ \dot{y} \ \dots \ y^{(n-1)}]^T$$

④ Write the coupled first-order differential equations -

$$\frac{d}{dt} x_1 = \frac{d}{dt} y = \dot{y} = x_2$$

$$\frac{d}{dt} x_2 = \frac{d}{dt} \dot{y} = \ddot{y} = x_3$$

...

$$\frac{d}{dt} x_n = \frac{d}{dt} y^{(n-1)} = g(y, \dot{y}, \dots, y^{(n-2)}, u) = g(x_1, x_2, \dots, x_n, u)$$

from governing
ODEs

⑤ Write system of first-ODEs as matrix.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \vdots \\ \dot{x}_n \end{bmatrix} = \begin{bmatrix} x_2 \\ x_3 \\ \vdots \\ g(x_1, x_2, \dots, x_n, u) \end{bmatrix}$$

$$\dot{x} = f(x, u)$$

Ex. ① Mass-Spring System $m\ddot{y}(t) + Ky(t) = u(t)$

① Identify $n=2$ (2nd order)

② Define States $x_1 = y, x_2 = \dot{y}$

③ Create the state vector $x = [x_1 \ x_2]^T = [y \ \dot{y}]^T$

④ Write the coupled first-ODEs:

$$\frac{d}{dt} x_1 = \frac{d}{dt} y = \dot{y} = x_2$$

$$\frac{d}{dt} x_2 = \frac{d}{dt} \dot{y} = \ddot{y} = \frac{u(t) - Ky(t)}{m} = \frac{u(t) - Kx_1}{m}$$

⑤ Write system of first-ODEs as matrix -

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} x_2 \\ \frac{u(t) - Kx_1}{m} \end{bmatrix}$$

This system is actually linear
in state's $x \times 1$ but u

This system is actually linear in state & inputs!
 So, we can write the eqns in the following manner.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ k_m & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u(t)$$

$$\dot{x} = Ax + Bu$$

Sol ② (Higher order system) Planar Quadrotor Model

$$m\ddot{y} = (\sin \phi) u_1$$

$$m\ddot{z} = (\cos \phi) u_2 + mg$$

$$I_{xx} \ddot{\phi} = u_2$$

① Identify $n=2$

② Define states $x_1 = y, x_2 = z, x_3 = \phi, x_4 = \dot{y}, x_5 = \dot{z}, x_6 = \ddot{\phi}$

③ Define the state vector

$$x = [x_1 \ x_2 \ x_3 \ x_4 \ x_5 \ x_6]^T = [y \ z \ \phi \ \dot{y} \ \dot{z} \ \ddot{\phi}]^T$$

④ Define the system of first-order diff' eqns:

$$\frac{d}{dt} x_1 = \frac{d}{dt} y = \dot{y} = x_4 \quad \frac{d}{dt} x_4 = \frac{d}{dt} \dot{y} = \ddot{y} = (\sin \phi) u_1 = \frac{(\sin x_3) u_1}{m}$$

$$\frac{d}{dt} x_2 = \frac{d}{dt} z = \dot{z} = x_5 \quad \frac{d}{dt} x_5 = \frac{d}{dt} \dot{z} = \ddot{z} = \frac{(\cos \phi) u_2 - g}{m}$$

$$\frac{d}{dt} x_3 = \frac{d}{dt} \phi = \dot{\phi} = x_6 \quad \frac{d}{dt} x_6 = \frac{d}{dt} \dot{\phi} = \ddot{\phi} = \frac{u_2}{I_{xx}}$$

⑤ Write system of first ODEs as matrices.

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \\ \dot{x}_5 \\ \dot{x}_6 \end{bmatrix} = \begin{bmatrix} x_4 \\ x_5 \\ x_6 \\ \frac{\sin(x_3) u_1}{m} \\ \frac{\cos(x_3) u_2 - g}{m} \\ u_2 \end{bmatrix}$$

These are nonlinear because of the sine & cosine of the state x_3 .