[4]

Determine:

(a) Volumetric efficiency on the air basis alone (b) Air-fuel ratio (c) The brake mean effective pressure (d) The relative efficiency on the brake thermal efficiency Q.5 i. Explain different types of brakes. ii. Explain the condition of self energizing condition for brakes. Describe the design steps for internal expanding brakes. Q.6 Attempt any two: Describe the design steps for friction clutch. 5 Explain the function of cone and centrifugal clutch. Describe the torque transmitting capacity of clutch. 5

Total No. of Questions: 6

Total No. of Printed Pages:4

Enrollment	No	••



Faculty of Engineering End Sem (Odd) Examination Dec-2022

AU3CO13 / ME3CO12 Machine Design -II

Kno	owledge is Power	Programme: B.Te	ch. E	Branch/Specialisation: AU/M	E
Durat	ion: 3 Hrs.			Maximum Marks: (60
Note:	(a) All questi	ions are compulsor	. Internal choices,	if any, are indicated. Answe	rs
	of Q.1 (M	CQs) should be wri	tten in full instead	of only a, b, c or d.	
(b) Us	e of design da	ata book is permitte	d & assume suitab	le data if required with reason	n.
Q.1 i	The jour	rnal bearings are ge	nerally used in	·	1
	(a) Belts	s (b) Columns	(c) Beams (d) Supports	
i	i. Which o	of the following is to	rue for the bearing'	's body?	1
		al moment of variou Il moments in 3D	s forces acting on	the body is the vector sum	
		al moment of various of all moments in 3	_	n the body is the algebraic	
		al moment of variou dimension	s forces acting on	the body is always zero in	
	•		s forces acting on	the body is the vector sum	
		ll moments which is the dimensions	s perpendicular to	each other forces whatever	
i	ii. The sho	ortest distance meas	ured along the no	rmal to the helix between	1
	correspo	onding points on the	adjacent teeth is c	called	
	(a) Gear	r pitch	(b) Helical pitch	1	
	(c) Circu	ular pitch	(d) Normal circu	ular pitch	
i	v. The ang	le at which the teet	n of the gear are in	clined to the axis of a gear	1
	is called	l as			
	(a) Pitch	n angle	(b) Normal angl	e	
	(c) Heli	x angle	(d) Gear angle		
•	v. A four-s	stroke diesel engine	operating at 1200	rpm uses 0.1 kg of fuel in	1
	5 minut	es and has the brak	e specific fuel con	sumption equals to 0.1910	
	_	h. Calculate the to	-		
	(a) 6.5 k	kg-m (b) 8 kg-m	(c) 7.5 kg-m	d) 5 kg-m	

P.T.O.

vi.	Rapid explosion of air-fuel mixture within the cylinder, when ignited by	1	
٧1.	a spark is known as	1	
	(a) Constant heat combustion		
	(b) Constant pressure combustion		
	•		
	(c) Constant volume combustion		
••	(d) Cut off ratio	1	
vii.	The energy absorb by brake is always	1	
	(a) Potential (b) Kinetic		
	(c) Either potential or kinetic (d) None of these	_	
viii.		1	
	the disk is 1m and time taken to come to stop the disk by brake is 1.6		
	sec. Square of radius of gyration is 0.2. Calculate the angle through		
	which disk rotated during braking period.		
	(a) 27.5 rad. (b) 24.6 rad. (c) 29.3 rad. (d) 32.4 rad.		
ix.	Clutch and coupling perform the same action.	1	
	(a) Both being permanent joints		
	(b) No they are different type of joints		
	(c) Both being temporary joints		
	(d) None of these		
х.	A plate clutch consists of 1 pair of contacting surfaces. The inner and	1	
	outer diameter of the friction disk is 100mm and 200mm respectively.		
	The coefficient of friction is 0.2 and permissible intensity of pressure is		
	1.5N/mm ² . Assuming uniform wear theory, calculate the operating force		
	in the clutch.		
	(a) 15546 N (b) 12344 N (c) 23562 N (d) 24543 N		
i.	Explain different type of lubricants.	2	
ii.	Describe the hydrodynamic theory.	3	
iii.	Explain different type of roller bearing and describe the bearing life.	5	
iv.	Describe the selection parameter of ball and roller bearings.	5	
i.	Explain the Lewis equation.	2	
ii.	Describe the design steps for straight tooth spur and helical gears.	8	
iii.	A gear drive consists of two gears, A and B, and has a velocity ratio of		
	1.50. Gear A, the smaller of the two gears, revolves at 126 rpm in the		
	clockwise direction, and has 28 teeth. If the gears have a module of		
	2 mm, determine the following:		

Q.2

OR

Q.3

OR

- (a) The number of teeth on Gear B
- (b) The pitch (reference) diameters for the two gears
- (c) The addendum
- (d) The dedendum
- (e) The circular pitch
- (f) The tooth thickness
- (g) The speed of Gear B

The theoretical centre distance of the two gears

Q.4 i. Write short note on any three design considerations in I. C. Engine

ii. A trial carried out in a four-stroke single cylinder gas engine gave the 7 following results. Cylinder diameter =300 mm, Engine stroke =500 mm, Clearance volume =6750 cc, Explosions per minute =100, P_{max} (kN/m²) = 765, Net work load on the brake =190 kg, Brake diameter =1.5 m Rope diameter =25mm, Speed of the engine =240rpm, Gas used =30 m³/kghr, Calorific value of gas =2.515 kJ/m³. Determine Compression ratio, Mechanical efficiency, Indicated thermal efficiency, Air standard efficiency, Relative efficiency, assume r =1.4

Design of Piston

Design a cast iron piston for a single acting four stroke diesel engine with the following data:

Cylinder bore = 300 mm

Length of stroke = 450 mm

Speed = 300 rpm

Indicated mean effective pressure = 0.85 Mpa

Maximum gas pressure = 5 Mpa

Fuel consumption = 0.3 kg per BP per h

Higher calorific value of fuel = 44000 kJ/kg

Assume suitable data, if necessary.

OR iii. The following observations are recorded during a test on a four-stroke 7 petrol engine, F.C = 3000 of fuel in 12sec, speed of the engine is 2500 rpm, B.P = 20 kW, Air intake orifice diameter = 35 mm, Pressure across the orifice = 140 mm of water coefficient of discharge of orifice = 0.6 , piston diameter = 150 mm, stroke length = 100 mm, Density of the fuel = 0.85 gm/cc , r =6.5, Cv of fuel = 42000 kJ/kg, Barometric pressure = 760mm of Hg , Room temperature = 24 °C.

P.T.O.

3

Marking Scheme

AU3CO13 / ME3CO12 Machine Design -II

Q.1	i.	(d) Supports	1 Mark	1
	ii.	(c) Total moment of various forces acting on the body is a	always zero in	1
		any dimension	1 Mark	
	iii.	(d) Normal circular pitch	1 Mark	1
	iv.	(c) Helix angle	1 Mark	1
	v.	(c) 7.5 kg-m	1 Mark	1
	vi.	(c) Constant volume combustion	1 Mark	1
	vii.	(c) Either potential or kinetic	1 Mark	1
	viii.	(c) 29.3 rad.	1 Mark	1
	ix.	(b) No they are different type of joints	1 Mark	1
	х.	(a) 15546 N	1 Mark	1
Q.2	i.	Different type of lubricants	2 Marks	2
Q.2	ii.	Explanation with diagram	3 Marks	3
	iii.	Type of roller bearing	2.5 Marks	5
	111.	Bearing life.	2.5 Marks	3
OR	i x/	Selection parameter of ball bearings	2.5 Marks	5
OK	17.	Selection parameter of roller bearings	2.5 Marks	3
		Selection parameter of folier bearings	2.3 Walks	
Q.3	i.	Lewis equation with notation	2 Marks	2
	ii.	Design Steps of spur gears	4 Marks	8
		Design steps of helical gears.	4 Marks	
OR	iii.	The number of teeth on Gear B	1 Mark	8
		The pitch (reference) diameters for the two gears	1 Mark	
		The addendum	1 Mark	
		The dedendum	1 Mark	
		The circular pitch	1 Mark	
		The tooth thickness	1 Mark	
		The tooth thickness	1 War	
		The speed of Gear B	1 Mark	
0.4	i	The speed of Gear B The theoretical centre distance of the two gears	1 Mark 1 Mark	3
Q.4	i.	The speed of Gear B	1 Mark 1 Mark 1 Mark each	3
Q.4		The speed of Gear B The theoretical centre distance of the two gears Three design considerations in I. C. Engine	1 Mark 1 Mark 1 Mark each (1 Mark*3)	
Q.4	i. ii.	The speed of Gear B The theoretical centre distance of the two gears Three design considerations in I. C. Engine Design formula	1 Mark 1 Mark 1 Mark each (1 Mark*3) 1 Mark	3
Q.4		The speed of Gear B The theoretical centre distance of the two gears Three design considerations in I. C. Engine Design formula Various important dimension	1 Mark 1 Mark 1 Mark each (1 Mark*3) 1 Mark 4 Marks	
Q.4 OR	ii.	The speed of Gear B The theoretical centre distance of the two gears Three design considerations in I. C. Engine Design formula	1 Mark 1 Mark 1 Mark each (1 Mark*3) 1 Mark	

		(b) Air-fuel ratio	1 Mark	
		(c) The brake mean effective pressure	2 Marks	
		(d) The relative efficiency on the brake thermal efficiency	2 Marks	
Q.5	i.	Any four types of brakes with explanation	1 Mark each	4
			(1 Mark*4)	
	ii.	Self energizing brakes	2 Marks	6
		Related diagram	2 Marks	
		Moment equation	2 Marks	
OR	iii.	Related diagram	1 Mark	6
		Design procedure	3 Marks	
		Related formula	2 Marks	
Q.6		Attempt any two:		
	i.	Design Steps	3 Marks	5
		Related formula	2 Marks	
	ii.	Function & related diagram for cone clutch	2.5 Marks	5
		Function & related diagram for centrifugal clutch	2.5 Marks	
	iii.	Torque transmitting capacity	2.5 Marks	5
		Various equation of torque	2.5 Marks	