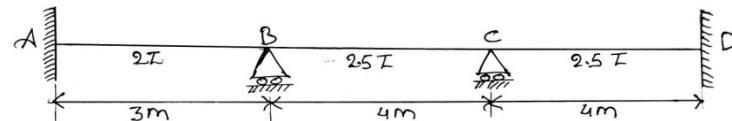
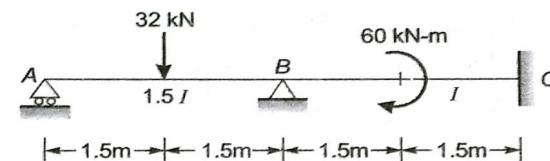


- ii. By MDM determine the support moment if support B settles by 5mm. Draw final bending moment diagram. Take $EI=45000 \text{ KN-M}$



7

- OR iii. By SDM Analysis the beam loaded as shown in figure below using slope deflection method. Draw the BMD



7

Q.6 Attempt any two:

- A three hinged arch has a span of 30 meters and a rise of 10 m. The arch carries a uniformly distributed load of 60 KN per meter on the left half of its span. It also carries two concentrated load of 60 KN and 100 KN at 5 m and 10 m from the right end. Determine the horizontal thrust, at each support.
- A cable carrying a load of 10KN per meter run of horizontal span is stretched between supports 100 meter apart. The supports are at the same level and the central dip is 8 meters. Find the greatest and the least tension in the cable.
- A two hinged parabolic arch of span 25 m and rise 5 m carries a uniformly distributed load of 40 KN/m over the left half of the span and a concentrated load of 100 KN at the crown. Find the horizontal thrust at each support.

5

5

5

Total No. of Questions: 6

Total No. of Printed Pages: 4

Enrollment No.....

Faculty of Engineering

End Sem (Even) Examination May-2018

CE3CO06 Structural Analysis -I

Programme: B.Tech.

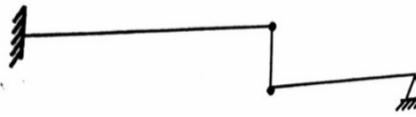
Branch/Specialisation: CE

Duration: 3 Hrs.

Maximum Marks: 60

Note: All questions are compulsory. Internal choices, if any, are indicated. Answers of Q.1 (MCQs) should be written in full instead of only a, b, c or d.

- Q.1 i. A pin jointed plane frame is unstable if 1
 (a) $(m+r) < 2j$ (b) $m+r = 2j$ (c) $(m+r) > 2j$ (d) None of these
 ii. The beam shown in the figure is 1

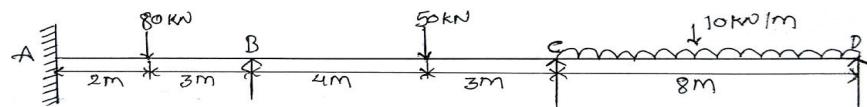


- (a) Determinate stable (b) Stable
 (c) Indeterminate (d) Unstable
- iii. When the length of a simply supported beam, carrying a point load at mid span is doubled the deflection below the load becomes 1
 (a) Two times (b) Four Times
 (c) Eight Times (d) None of these
- iv. Bending Moment at any section X of the cantilever beam 1
 (a) $M = P^2 X$ (b) $M = -P X$ (c) $M = P X^2$ (d) $M = -P^2 X^2$
- v. The ordinates of influence line diagram for bending moment always have the dimensions of 1
 (a) Force (b) Length
 (c) Force x Length (d) Force / Length
- vi. Influence line for horizontal thrust in a two hinged parabolic arch is 1
 (a) Parabolic (b) Cubic (c) Triangular (d) Rectangular
- vii. Slope deflection method can be used for analysing 1
 (a) Statically determinate structure
 (b) Statically indeterminate structure
 (c) Unstable structure
 (d) None of these

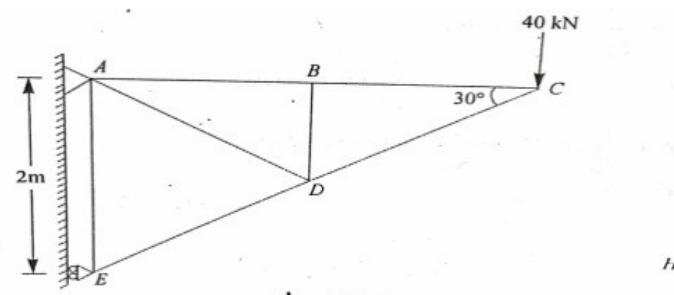
[2]

- viii. Moment distribution method is nothing but a
 - (a) Lateral translation method
 - (b) Displacement method
 - (c) Rotational method
 - (d) None of these
- ix. In a solid arch, shear force acts
 - (a) Vertically upward
 - (b) Along the axis of the arch
 - (c) Perpendicular to the axis of arch
 - (d) Tangentially to the arch
- x. The horizontal component of reaction of three hinged arch is called
 - (a) Horizontal thrust
 - (b) Vertical thrust
 - (c) Inplane thrust
 - (d) None of these

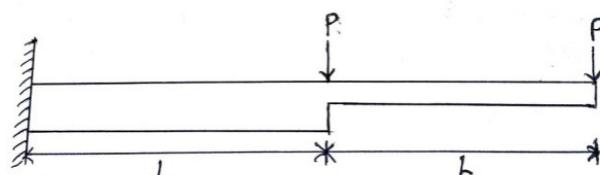
- Q.2 i. Define kinematic indeterminacy.
ii. Analyse beam by three moment equation.



- OR iii. Find the vertical and horizontal deflection of the joint C of the loaded truss shown in figure. The cross sectional areas of member CD and DE are each 2500 mm^2 and those of the other member are each 1250 mm^2 . Take $E = 200 \text{ KN/mm}^2$



- Q.3 i. Define principle of virtual work.
ii. Determine the deflection and rotation at the free end of the cantilever. Use unit load method.



1

1

1

2

8

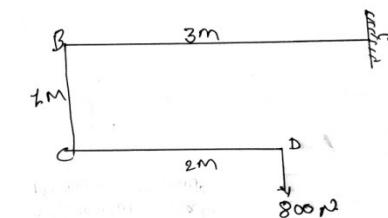
8

2

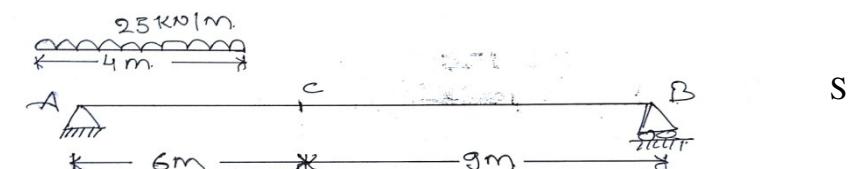
8

[3] 8

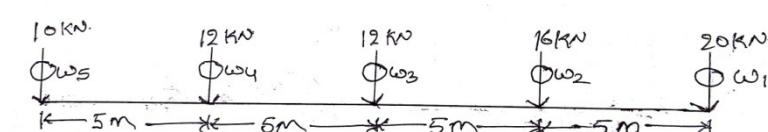
- OR iii. A mild steel bar 100mm diameter is bent as shown in fig. It is fixed horizontally at A and a load of 800 N hangs at D. Draw the bending moment diagram for the parts AB, BC, and CD indicating the maximum values. Find the maximum bending stress. Find also the deflection at D. Take $E = 2 \times 10^5 \text{ N/mm}^2$



- Q.4 i. Write down application of ILD.
ii. A simply supported beam has a span of 15m. A uniformly distributed load of 25KN/m and 4m long crosses the girder from left to right. Drawn the influence line diagram for shear force and bending moment at a section 6m from the left support. Also calculate maximum shear force and bending moment at this section



- OR iii. Five point load of 10KN, 12KN, 12KN, 16KN, and 20KN, spaced at 5m centre to centre rolls over a simply supported girder of 80m. The loads move left to right with 20KN load leading, then calculate position and magnitude maximum bending moment which may occur anywhere on the girder



- Q.5 i. Define Stiffness, Distribution Factor, Carry over Factor. 3

P.T.O.

Marking Scheme
CE3CO06 Structural Analysis -I

Question: - 1 objective (10 Marks)

(i) A pin jointed plane frame is unstable if

Ans. $(m + r) < 2j$

(ii) The beam shown in the figure is

Ans. Determinate stable

(iii) When the length of a simply supported beam, carrying a point load at mid span is doubled the deflection below the load becomes

Ans. Eight Times

(iv) Bending Moment at any section X of the cantilever beam

Ans. $M = -PX$

(v) The ordinates of influence line diagram for bending moment always have the dimensions of

Ans. Length

(vi) Influence line for horizontal thrust in a two hinged parabolic arch is

Ans. Cubic

(vii) Slope deflection method can be used for analysing

Ans. Statically indeterminate structure

(viii) Moment distribution method is nothing but a

Ans. Displacement method

(ix) In a solid arch, shear force acts

Ans. Perpendicular to the axis of arch

(x) The horizontal component of reaction of three hinged arch is called

Ans. Horizontal thrust

Question 2 (2, 8, or 8 Marks)

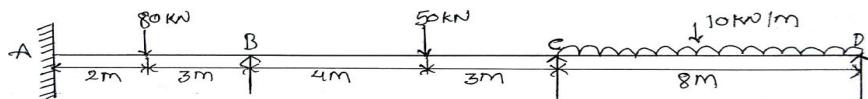
(i) Define kinematic indeterminacy.

Ans.

KINEMATIC INDETERMINACY

"The degree of kinematic Indeterminacy may be defined as the total number of degrees of freedom (independent displacement co-ordinates) at various joints in a skeletal structure".

(ii) Analyse beam by three moment equation.



$$10M_A + 5M_B - 640 = 0 \quad [mark] \\ MA + 0.5MB = 64 \quad [0.5]$$

$$0.5 \text{ mark} \quad [MA + 4.8MB + 1.4Mc = 306]$$

$$0.5 \text{ mark} \quad [MB + 4.28 Mc = 317.6]$$

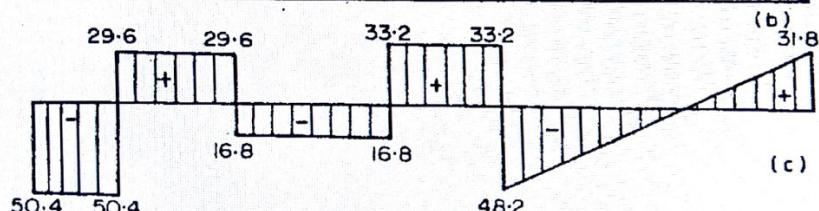
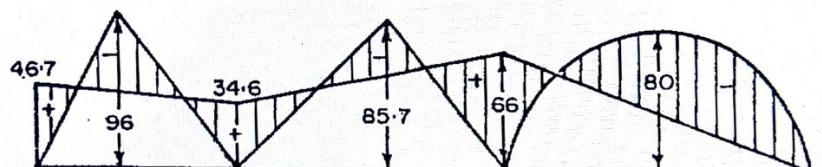
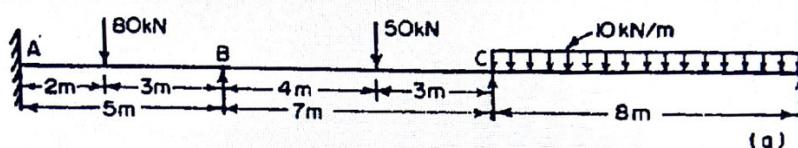
$$0.5 \text{ mark} \quad [MB + 0.325 Mc = 56.2]$$

$$+ \text{mark} \quad [MB = 34.6 \text{ KN-m} \\ MA = 46.7 \text{ KN-m}]$$

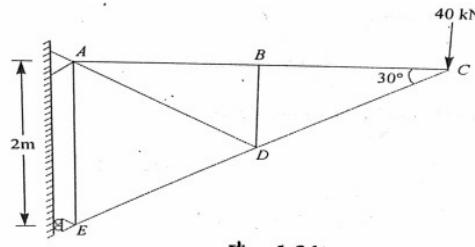
Reaction

$$0.5 \text{ mark each.} \quad R_D = 31.8 \text{ kN} \\ R_C = 81.4 \text{ kN} \\ R_A = 50.4 \text{ kN} \\ R_B = 46.4 \text{ kN}$$

B.M. S.F. diagram have
1.5 marks each.



(iii) Find the vertical and horizontal deflection of the joint C of the loaded truss shown in figure. The cross sectional areas of member CD and DE are each 2500 mm^2 and those of the other member are each 1250 mm^2 . Take $E = 200 \text{ KN/mm}^2$



Ans. $H_e, V_a, H_a, P_{ab}, P_{bc}, P_{cd}, P_{de}, P_{ae}, P_{ad}, P_{bd}$, = 0.5 mark each

Vertical deflection, Horizontal deflection = 0.5 mark each

Table = 1 mark

Diagram with force = 1 mark

STRAIN ENERGY

We have,

$$\text{Taking moments about } A, \quad AC = 2 \cos 30^\circ = 2\sqrt{3} \text{ metre}$$

$$H_e \times 2 = 40 \times 2\sqrt{3}$$

$$H_e = 40\sqrt{3} \text{ kN} \rightarrow$$

$$V_o = 40 \text{ kN} \uparrow$$

$$H_a = 40\sqrt{3} \text{ kN} \leftarrow$$

Joint C. Resolving vertically,
 $P_{cd} \sin 30^\circ = 40$

$$\therefore P_{cd} = 80 \text{ kN (compressive)}$$

Resolving horizontally,

$$P_{cd} = 80 \cos 30^\circ = 40\sqrt{3} \text{ kN (tensile)}$$

Joint B. $P_{bd} = 0; P_{ba} = 40\sqrt{3} \text{ kN (tensile)}$

Joint D. $P_{da} = 0; P_{de} = 80 \text{ kN (compressive)}$

Joint E. Resolving vertically,

$$P_{ea} = 80 \sin 30^\circ \\ = 40 \text{ kN (tensile)}$$

To find the vertical deflection at C

Now remove the given load system and apply a vertical load of 1 kN at C. The forces in the members of the frame for this case will be $\frac{1}{40}$ of the forces in the previous case. See Fig. 1.86.

$$\text{Vertical deflection of } C = y = \sum \frac{PKI}{AE}$$

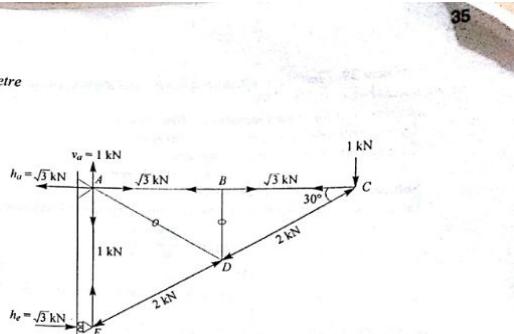


Fig. 1.86

Member	$P (\text{kN})$	$K (\text{kN})$	$I (\text{mm})$	$A (\text{mm}^2)$	$\frac{PKI}{A}$
AB	$-40\sqrt{3}$	$-\sqrt{3}$	$1000\sqrt{3}$	1250	166.3
BC	$-40\sqrt{3}$	$-\sqrt{3}$	$1000\sqrt{3}$	1250	166.3
CD	80	2	2000	2500	128
DE	80	2	2000	2500	128
AE	-40	-1	2000	1250	64
AD	0	0	2000	1250	0
BD	0	0	1000	1250	0
Total					652.6

∴ Vertical deflection of C

$$= y = \sum \frac{PKI}{AE} = \frac{652.6}{200} \\ = 3.263 \text{ mm}$$

To find the horizontal deflection of C.

In this case apply a unit load i.e., 1 kN horizontally at C. The forces in the members for this loading are shown in Fig. 1.87.

$$\text{Horizontal deflection of } C = \sum \frac{PKI}{AE}$$

$$= \frac{2(-40\sqrt{3})(-1)1000\sqrt{3}}{1250 \times 200} = 0.96 \text{ mm}$$

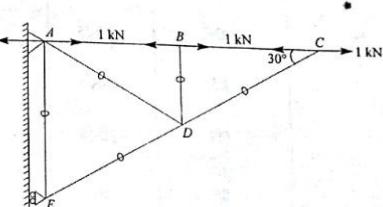


Fig. 1.87

Question: - 3 (2, 8, or 8 Marks)

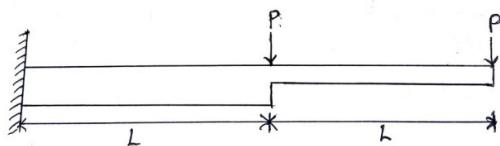
(i) Define principle of virtual work.

Ans: - The principle of virtual work state that in equilibrium the virtual work of the force applied to a system is zero.

or

In other word external work of system equal to internal work of system.

(ii) Determine the deflection and rotation at the free end of the cantilever. Use unit load method.



Ans.

Solution:

For given beam,

Deflection at free end is given by

$$\Delta = \sum \frac{Mm_1}{EI} dx \rightarrow 1 \text{ mark}$$

$$\begin{aligned} \Delta &= \int_0^L \frac{(-Px)(-x)}{EI} dx + \int_0^L \frac{[P(L+x)+Px](x+L)}{E(2I)} dx \\ &= \int_0^L \frac{Px^2 dx}{EI} + \int_0^L \frac{[P(x+L)^2 + Px(x+L)] dx}{2EI} \\ &= \int_0^L \frac{Px^2 dx}{EI} + \int_0^L \frac{P}{2EI} [x^2 + L^2 + 2Lx + x^2 + Lx] dx \end{aligned}$$

2 marks

Portion	CB	BA
Origin	C	B
Limit	O-L	O-L
M	-Px	-P(L+x)+Px
m ₁	-x	-(x+L)
m ₂	-1	-1
MOI	I	2I

$$= \int_0^L \frac{Px^2 dx}{EI} + \int_0^L \frac{P}{2EI} [2x^2 + 3Lx + L^2] dx$$

$$= \frac{P}{2EI} \left[\int_0^L 2x^2 dx + \int_0^L (2x^2 + 3Lx + L^2) dx \right]$$

$$= \frac{P}{2EI} \left[2 \left[\frac{x^3}{3} \right]_0^L + \left[2 \cdot \frac{x^3}{3} + 3 \cdot \frac{Lx^2}{2} + L^2 \cdot x \right]_0^L \right]$$

$$= \frac{P}{2EI} \left[2 \cdot \frac{L^3}{3} + \frac{2}{3} L^3 + \frac{3}{2} L^3 + L^3 \right]$$

$$\Delta = \frac{23PL^3}{12EI} \rightarrow 1 \text{ mark}$$

Rotation at free end is given by,

$$\theta = \sum \frac{Mm_2}{EI} dx \rightarrow 1 \text{ mark}$$

$$= \int_0^L \frac{(-Px)(-1)dx}{EI} + \int_0^L \frac{[-P(L+x)+Px](-1)dx}{2EI}$$

$$= \int_0^L \frac{Px dx}{EI} + \int_0^L \frac{[P(L+x)+Px]}{2EI} dx$$

$$= \frac{P}{2EI} \left[\int_0^L 2x dx + \int_0^L (2x+L) dx \right]$$

$$= \frac{P}{2EI} \left[2 \left[\frac{x^2}{2} \right]_0^L + 2 \left[\frac{x^2}{2} \right]_0^L + L[x]_0^L \right]$$

1 mark

$$\theta = \frac{P}{2EI} \left[2 \cdot \frac{L^2}{2} + 2 \cdot \frac{L^2}{2} + L \cdot L \right] = \frac{3PL^2}{2EI} \rightarrow 1 \text{ mark}$$

(iii) A mild steel bar 100mm diameter is bent as shown in fig. it is fixed horizontally at A and a load of 800 N hangs at D. Draw the bending moment diagram for the parts AB, BC, and CD indicating the maximum values. Find the maximum bending stress. Find also the deflection at D. Take $E = 2 \times 10^5$ N/mm²

Ans: - B.M at D = 0

B.M at C (CD) = -1600 Nm

B.M.= 0.5 mark for each

B.M. at C (CB) = 1600 Nm

Vertical deflection = 0.5 mark

B.M. at B (AB) = 1600 Nm

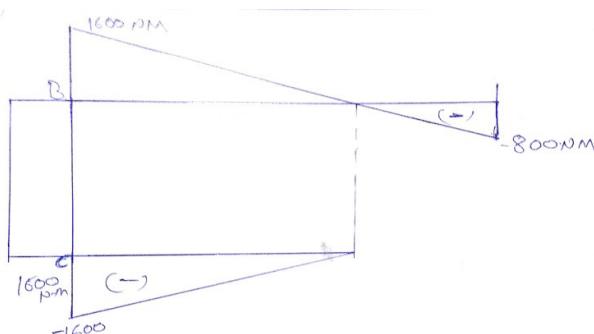
Maximum bending stress = 2 Marks

B.M. at A = -800 Nm

Maximum bending stress = $(M/Z) = (1600 / 98174.7704) = 16.29 \text{ N/mm}^2$

Strain energy stored by bar = Summation of $\int M^2 ds / 2EI$ **1 Mark for formula**
 $= 29/6 \times P^2/EI$

Vertical deflection at D = 4.92 mm



2 Marks for B.M. Diagram

Question: - 4 (3, 7, or 7 Marks)

(i) Write down application of ILD

1 mark of each point

2.4 Application of ILD

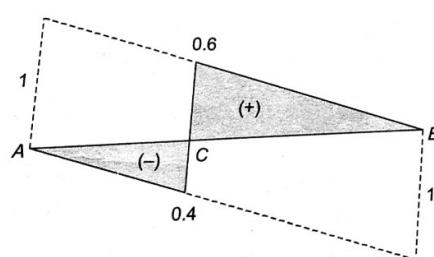
- ILD can be used to study the effect of a moving load on the structure.
- ILD can be used to find position of live load which will produce maximum value of a particular stress function.
- ILD can be used to calculate total value of a particular stress function due to a multiple load system for e.g.

Net BM at C due to load system will be given by
 $M_C = P_1 y_D + P_2 y_E + w \times \text{area of ILD below UDL}$.

Question 4 (ii)

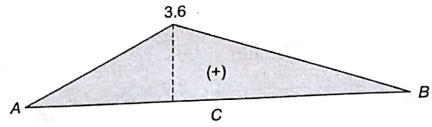
Solution:

ILD for SF at C:



1.5 marks

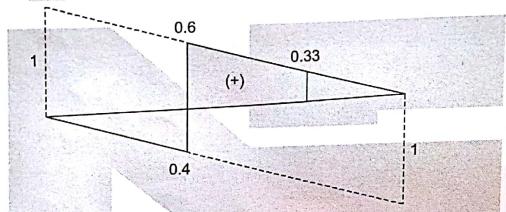
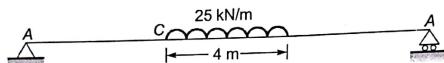
ILD for BM at C:



1.5 marks

$$\text{Ordinate of BM at section } = \frac{a(L-a)}{L} = \frac{6 \times 9}{15} = 3.6 \text{ units}$$

(i) Maximum Positive Shear Force at Section

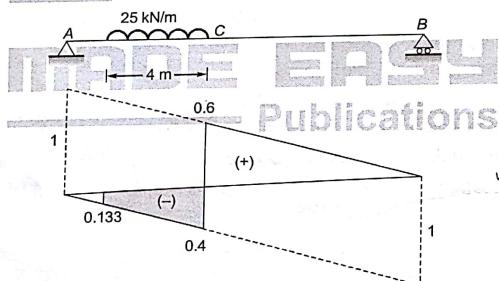


1.5 marks

$$\therefore \text{Maximum positive SF at } C = \text{Area of ILD below UDL} \times w$$

$$= \frac{1}{2} \times (0.6 + 0.33) \times 4 \times 25 = 46.67 \text{ kN}$$

(ii) Maximum Negative SF at C:



1.5 marks

$$\therefore \text{Maximum negative SF at } C = \text{Area of ILD below UDL} \times w$$

$$= \frac{1}{2} \times (0.4 + 0.133) \times 4 \times 25 = 26.65 \text{ kN}$$

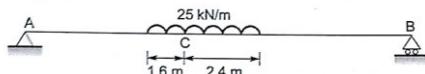
(iii) Maximum BM at C:

Maximum bending moment at C will occur when average loading just to the left of C is equal to average loading just to right of C.

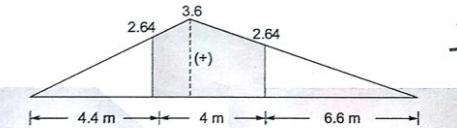
$$\frac{a}{a'} = \frac{L-a}{L'-a'}$$

$$\frac{6}{a'} = \frac{9}{4-a'}$$

$$a' = 1.6 \text{ m}$$



1.5 marks



$$\therefore \text{Maximum bending moment at } C = \text{Area of ILD below loading} \times w$$

$$= \left[(2.64 \times 4) + \left(\frac{1}{2} \times 4 \times 0.96 \right) \right] \times 25$$

$$= (10.56 + 1.92) \times 25 = 312 \text{ kN-m}$$

Maximum B.M. at C carries only 1 mark

Question 4 (iii)

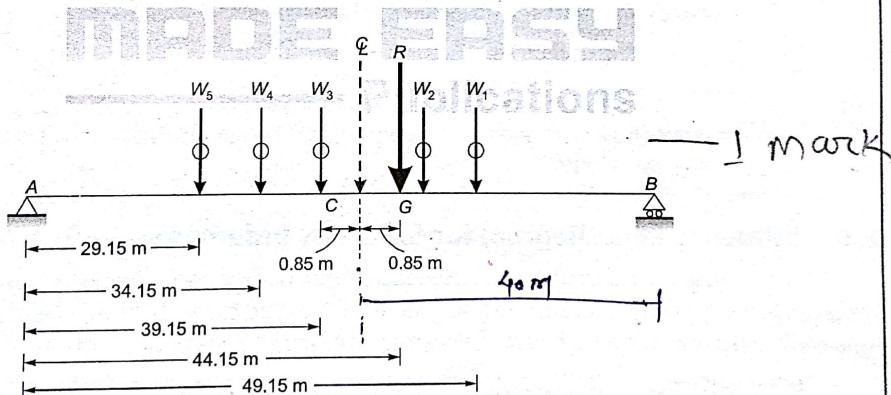
Solution:

Position of CG of load system,

$$\bar{X} = \frac{10 \times 0 + 12 \times 5 + 12 \times 10 + 16 \times 15 + 20 \times 20}{10 + 12 + 12 + 16 + 20} \rightarrow 1 \text{ mark}$$

$$= 11.71 \text{ m} \quad (\text{From } 10 \text{ kN load})$$

Case-1: Let W_3 is load under consideration, hence maximum bending moment will occur below W_3 which is nearest to CG for maximum bending moment under W_3 , load system should occupy such a position that the center of span is midway between CG of load system and load under consideration i.e. W_3 .



$$\Sigma M_A = 0$$

$$R_B \times 80 - 20 \times 49.15 - 16 \times 44.15 - 12 \times 39.15 - 12 \times 34.15 - 10 \times 29.15 = 0$$

$$R_B = 35.75 \text{ kN}$$

$$R_A = 70 - 35.75 = 34.25 \text{ kN}$$

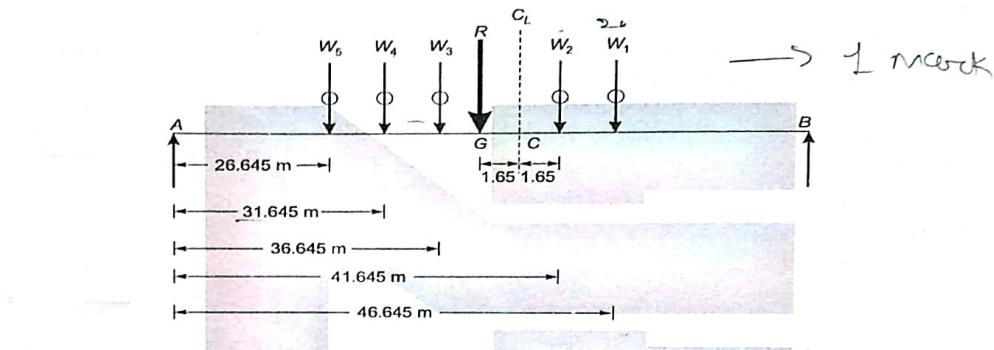
→ 1 mark

Absolute maximum bending moment = Bending moment under load W_3
 $= R_A \times 39.15 - 10 \times 10 - 12 \times 5$
 $= 1180.80 \text{ kN-m}$ → 1 mark

Case-2: Let W_2 is load under consideration.

W_2 is heavier than W_3 , however W_3 is nearest to C.G. Then maximum bending moment may occur below W_2 .

Let's consider maximum BM occurs below W_2 when the load system occupy such a position that the center of span is midway between CG of load system and load under consideration i.e. W_2 .



$$\Sigma M_A = 0 \quad 12 \times 31.645$$

$$R_A \times 80 - 20 \times 46.645 - 16 \times 41.645 - 12 \times 36.645 - 10 \times 26.645 = 0$$

$$R_B = 33.56 \text{ kN}$$

$$R_A = 36.44 \text{ kN}$$

→ 1 mark

Absolute maximum bending moment = Bending moment below load W_2
 $= R_A \times 41.645 - 10 \times 15 - 12 \times 10 - 12 \times 5$
 $= 36.44 \times 41.645 - 150 - 120 - 60$
 $= 1187.54 \text{ kN-m}$ → 1 mark

Hence absolute maximum bending moment is 1187.5 kN-m which occurs at a distance of 41.645 m from left support A under load W_2 .

Question: - 5 (3, 7, or 7 Marks)

(i) Define Stiffness, Distribution Factor, Carry over Factor.

Stiffness:— Stiffness for a member at a joint is the moment (force) required to produce unit rotation (displacement) at the joint.

1 marks
Stiffness depend upon the condition and properties of cross-section.

Distribution factor

Distribution factor for a member at a joint is the ratio of the stiffness (or relative stiffness) of the member to the total stiffness (or total relative stiffness).

$$DF = \frac{\text{Stiffness of the member}}{\text{Total stiffness of the joint.}}$$

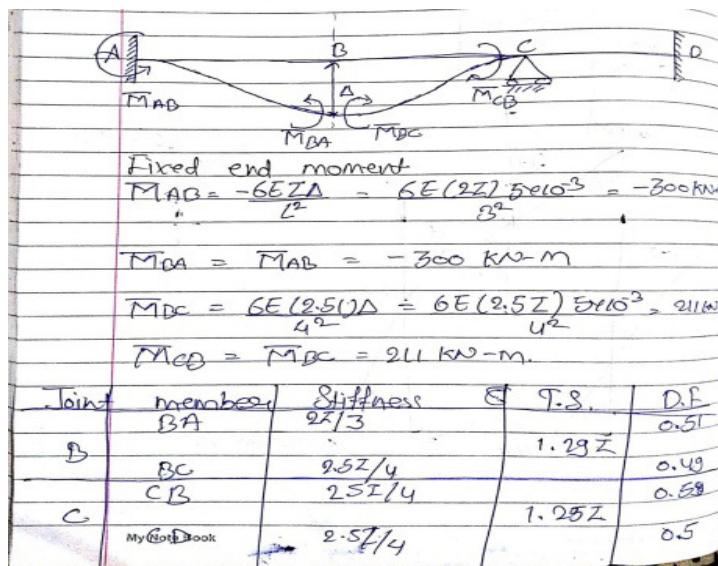
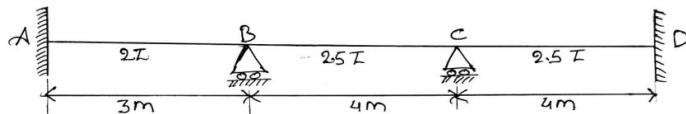
Carry over factors

It is define as the ratio of moment at the fixed end to the moment at the rotating near end.

$$CO = \frac{\text{Carry over moment at free end}}{\text{Applied moment at near end}}$$

$$f = \text{mark}$$

(ii) By MDM determine the support moment if support B settles by 5mm. Draw final bending moment diagram. Take $EI=45000 \text{ KN-M}$



		0.51	0.49		0.5	0.5	
F.E.M.	-300	-300	211	211	0	0	
B.M.		45.39	43.61	-105.5	-105.5		
Com	22.70		-52.75	21.80		-52.75	
B.M.		+96.91	25.84	-10.9	-10.9		
Com	-13.45		-5.45	12.92		-5.45	
B.M.		+2.78	+2.67	-6.46	-6.46		
Com	1.39		-3.23	1.335		-3.23	
		-7.65	+1.58	-0.667	-0.667		
		0.825		-3.23	1.335		-0.333
		0.085	0.17	0.163	-0.667	-0.395	
				-0.187	0.081		-0.187
			+0.099	0.098	-0.395	-0.04	
F.E.M.	-261.55	-223	+223	123.96	-123.96	-61.96	

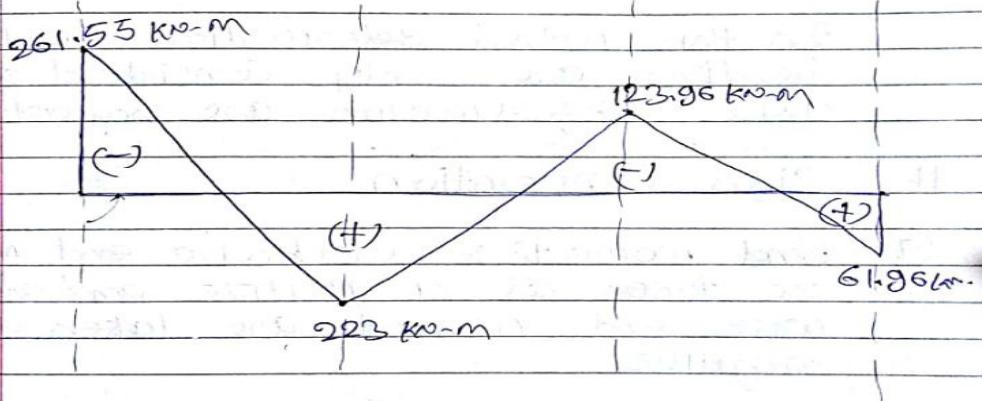


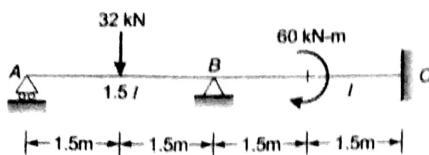
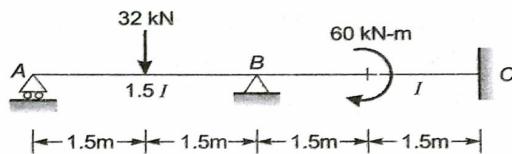
Diagram \rightarrow 4
Fixed end moment \rightarrow 2

Table D.F. \rightarrow 4

F.E.M. table \rightarrow 2

B.M. Diagram \rightarrow 1

(iii) By SDM Analysis the beam loaded as shown in figure below using slope deflection method.
Draw the BMD



Solution:

Fixed end moments:

0.25 mark
each.

$$\bar{M}_{AB} = -\frac{PL}{8} = -\frac{32 \times 3}{8} = -12 \text{ kN-m}$$

$$\bar{M}_{BA} = +\frac{PL}{8} = +12 \text{ kN-m}$$

$$\bar{M}_{BC} = +\frac{M_0}{4} = +\frac{60}{4} = +15 \text{ kN-m}$$

$$\bar{M}_{CB} = +\frac{M_0}{4} = +15 \text{ kN-m}$$

Slope deflection equations:

Member AB:

$$M_{AB} = \bar{M}_{AB} + \frac{2EI(1.5I)}{L} \left(2\theta_A + \theta_B - \frac{3\Delta}{L} \right)$$

$$M_{AB} = -12 + \frac{3EI}{3} (2\theta_A + \theta_B)$$

0.5 mark $\leftarrow M_{AB} = -12 + 2EI\theta_A + EI\theta_B \quad \dots(i)$

and

$$M_{BA} = \bar{M}_{BA} + \frac{2EI(1.5I)}{L} \left(2\theta_B + \theta_A - \frac{3\Delta}{L} \right)$$

0.5 mark $\leftarrow M_{BA} = +12 + \frac{3EI}{3} (2\theta_B + \theta_A) \quad \dots(ii)$

Member BC:

$$M_{BC} = \bar{M}_{BC} + \frac{2EI(I)}{L} \left(2\theta_B + \theta_C - \frac{3\Delta}{L} \right) = +15 + \frac{2EI}{3} (2\theta_B - 0)$$

0.5 mark $\leftarrow M_{BC} = 15 + \frac{4EI\theta_B}{3} \quad \dots(iii)$

$$M_{CB} = \bar{M}_{CB} + \frac{2EI}{L} \left(2\theta_C + \theta_B - \frac{3\Delta}{L} \right) = +15 + \frac{2EI}{3} (0 + \theta_B)$$

0.5 mark $\leftarrow M_{CB} = +15 + \frac{2}{3} EI\theta_B \quad \dots(iv)$

Equilibrium equations:

There are two rotational unknowns θ_A and θ_B . Hence two joint equilibrium conditions are required.

$$M_A = 0$$

$$-12 + 2EI\theta_A + EI\theta_B = 0$$

Consider joint equilibrium at B,

$$M_{BA} + M_{BC} = 0$$

$$+12 + 2EI\theta_B + EI\theta_A + 15 + \frac{4}{3}EI\theta_B = 0$$

$$27 + 3.33EI\theta_B + EI\theta_A = 0$$

$$EI\theta_A + 3.33EI\theta_B = -27$$

On solving equation (A) and (B), we get

$$EI\theta_A = 11.823$$

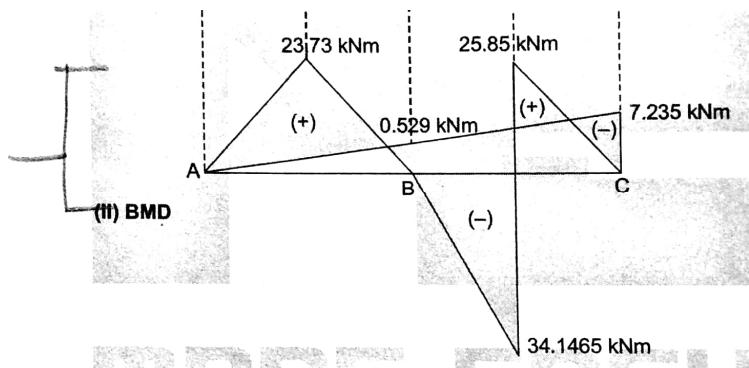
$$EI\theta_B = -11.647$$

Final end moments

0.25 mark each

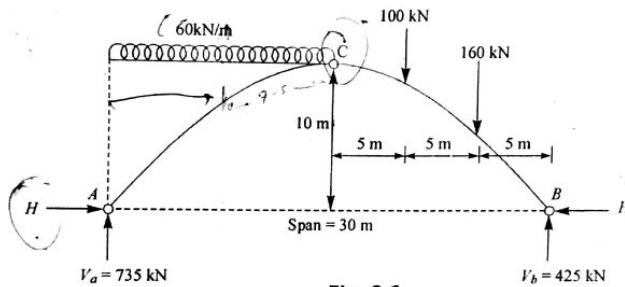
$$\begin{aligned} M_{AB} &= 0 \\ M_{BA} &= 12 + 2EI\theta_B + EI\theta_A \\ &= 12 + 2(-11.647) + 11.823 = 0.529 \text{ kNm} \\ M_{BC} &= 15 + \frac{4}{3}EI\theta_B = 15 + \frac{4}{3}(-11.647) = -0.529 \text{ kNm} \\ M_{CB} &= 15 + \frac{2}{3}EI\theta_B = 15 + \frac{2}{3}(-11.647) = 7.235 \text{ kNm} \end{aligned}$$

1 mark.



Question: - 6 Attempt any two: (5, 5, or 5 Marks)

- (i) A three hinged arch has a span of 30 meters and a rise of 10 m. The arch carries a uniformly distributed load of 60 KN per meter on the left half of its span. It also carries two concentrated load of 60 KN and 100 KN at 5 m and 10 m from the right end. Determine the horizontal thrust, at each support.



Solution. Let V_a and V_b be the vertical reactions at the supports A and B. Let H be the horizontal thrust.

Taking moments about the left end A, we have,

$$V_b \times 30 = 60 \times 15 \times \frac{15}{2} + 100 \times 20 + 160 \times 25$$

marks

Diagram \rightarrow 2

$V_a = 718.33 \text{ KN} \rightarrow 1$

$V_b = 341.67 \text{ KN} \rightarrow 1$

$H = 402.45 \text{ KN} \rightarrow 1$

Taking moments about C of the forces on the left hand side of C, we have,

$$735 \times 15 = 60 \times 15 \times \frac{15}{2} + H \times 10; \therefore H = 427.5 \text{ kN.}$$

..... about C of the forces on the right hand side of C, we have

(ii) A cable carrying a load of 10KN per meter run of horizontal span is stretched between supports 100 meter apart. The supports are at the same level and the central dip is 8 meters. Find the greatest and the least tension in the cable.

Solution. Each vertical reaction $V = \frac{wl}{2} = \frac{10 \times 100}{2} = 500 \text{ kN}$ $\rightarrow 1 \text{ mark}$ Diagram $\rightarrow 1$

$$\text{Horizontal reaction} = H = \frac{wl^2}{8h} = \frac{10 \times 100^2}{8 \times 8} \text{ kN} = 1562.5 \text{ kN} \rightarrow 1 \text{ mark}$$

$$\text{Maximum tension} = T_{\max} = \sqrt{V^2 + H^2} = \sqrt{500^2 + 1562.5^2} = 1640.55 \text{ kN} \quad 1 \text{- mark}$$

$$\text{Minimum tension} = T_{\min} = H = 1562.5 \text{ kN} \quad 1 \text{- mark}$$

(iii) A two hinged parabolic arch of span 25 m and rise 5 m carries a uniformly distributed load of 40 KN/m over the left half of the span and a concentrated load of 100 KN at the crown. Find the horizontal thrust at each support.

2 Marks

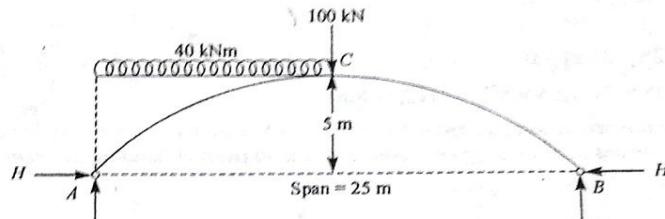


Fig. 2.84

Solution. Horizontal thrust due to uniformly distributed load on the left half of the span.

$$H_1 = \frac{wl^2}{16h} = \frac{40 \times 25^2}{16 \times 5} = 312.5 \text{ kN} \rightarrow 1 \text{ - marks}$$

Horizontal thrust due to point load at the crown,

$$H_2 = \frac{25}{128} \frac{wl}{h} = \frac{25}{128} \times \frac{100 \times 25}{5} = 97.66 \text{ kN} \rightarrow 1 \text{ mark}$$

Total horizontal thrust,

$$H = H_1 + H_2 = 312.50 + 97.66 = 410.16 \text{ kN} \rightarrow 1 \text{ mark}$$