# ENSC 483 - Project 1 Magnetic Levitation System Analysis

Dan Hendry (danh@sfu.ca), 301133878 Veronica Cojocaru (vca5@sfu.ca), 301055896

Simon Fraser University
School of Engineering Science

April 12, 2010

Course Instructor: Professor Mahdi Alavi

# Contents

	Contents			
1	System Description			
	1.1	Configuration	1	
	1.2	State Space Model - Inputs, Outputs and State Variables	2	
		Inputs	2	
		State Variables	2	
		Output	2	
2	Nonlinear Model			
	2.1	System Dynamic	3	
	2.2	State Space Model	4	
3	Linearized Model			
	3.1	Equilibrium Point	5	
	3.2	Linearizing About the Equilibrium Point	5	
		Modal Matrix	5	
		Input Matrix	6	
		Output Matrix	6	
	3.3	Linearized System Model	7	
4	Open Loop Analysis			
	4.1	Stability	8	
	4.2	Controllability	9	

4.5 Observability		J
References	1	ın

## 1

# System Description

## 1.1 Configuration

The magnetic levitation system is considered in its SISO (single input, single output) system configuration as it levitates a magnet via repulsion. As such, only the bottom coil and magnet are considered as shown in the simplified diagram 1.1.

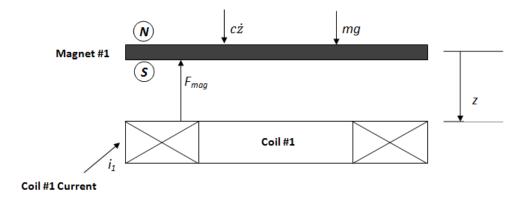


Figure 1.1: System Diagram

In this model, there are 3 forces acting on the magnet: the force of gravity (mg), the force of friction  $(c\dot{z})$  and the magnetic force form the drive coil  $(F_{mag})$ .

# 1.2 State Space Model - Inputs, Outputs and State Variables

For the system modeling done in this report, y denotes the output variable, u denotes the input variable and the vector  $\mathbf{x}$  denotes the state variables.

#### Inputs

The input to the system is the current in Coil #1, denoted in diagram 1.1 by  $i_1$ . This is given in equation 1.1.

$$\mathbf{u} = \left[ i_1 \right] \tag{1.1}$$

#### State Variables

The state variables for the system are chosen to be the magnets velocity  $(\dot{z})$  and its position (z). This is shown in equation 1.2

$$\mathbf{x} = \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = \begin{bmatrix} z \\ \dot{z} \end{bmatrix} \tag{1.2}$$

#### Output

The output of the system is the position of the disk magnet, denoted in diagram 1.1 by z. This is shown in equation 1.3:

$$\mathbf{y} = \left[ \begin{array}{c} z \end{array} \right] \tag{1.3}$$

# Nonlinear Model

#### 2.1 System Dynamic

From figure 1.1, summing the forces yields equation 2.1

$$m\ddot{z} = F_{mag} - c\dot{z} - mg \tag{2.1}$$

The force  $F_{mag}$  is given by 2.2 as presented by the manual [1]. The constants a, b, and N are empirically determined properties of the system. For the purposes of this report their values are assumed based on a similar system [2] and given in table 2.1.

$$F_{mag} = \frac{i}{a\left(z+b\right)^N} \tag{2.2}$$

Table 2.1: System properties [2]

a	0.000105
b	6.2
N	4

It is assumed friction and air resistance are negligible, so c=0 in equation 2.1. As such, equations 2.1 and 2.2 give equation 2.3

$$m\ddot{z} = \frac{i_1}{0.000105 (z + 6.2)^4} - 9.81 m \tag{2.3}$$

## 2.2 State Space Model

Using the state variables defined in section 1.2, two system equations may be written. These are presented in equations 2.4 and 2.5.

$$\dot{x_1} = f_1 = x_2 \tag{2.4}$$

$$\dot{x}_2 = f_2 = \frac{u}{0.000105 \, m \, (x_1 + 6.2)^4} - 9.81$$
 (2.5)

# Linearized Model

## 3.1 Equilibrium Point

The system is linearized about an equilibrium point. This point is given by equation 3.1 where  $z_0$  represents the initial position of the magnet. The input equilibrium may be found using equation 2.5 as shown in 3.3.

$$\mathbf{x}^{\star} = \begin{bmatrix} z_0 \\ 0 \end{bmatrix} \tag{3.1}$$

$$\frac{u^*}{0.000105 (z_0 + 6.2)^4} = 9.81 m \tag{3.2}$$

$$u^* = 0.00103005 \, m \left( z_0 + 6.2 \right)^4 \tag{3.3}$$

#### 3.2 Linearizing About the Equilibrium Point

#### **Modal Matrix**

Equations  $f_1$  and  $f_2$  from 2.4 and 2.5 may be used to obtain a linearized modal matrix. The modal matrix general form is given in equation 3.4. Using equations 3.5, this gives 3.6.

$$\mathbf{A} = \begin{bmatrix} \frac{df_1}{dx_1} & \frac{df_1}{dx_2} \\ \frac{df_2}{dx_1} & \frac{df_2}{dx_2} \end{bmatrix} \Big|_{x^*, u^*}$$
(3.4)

$$\frac{df_1}{dx_1} = 0 \qquad \frac{df_1}{dx_2} = 1 \qquad \frac{df_2}{dx_1} = -\frac{38095.23810 \, u}{m \left(x_1 + 6.2\right)^5} \qquad \frac{df_2}{dx_2} = 0 \tag{3.5}$$

$$\mathbf{A} = \begin{bmatrix} 0 & 1 \\ -\frac{39.24}{z_0 + 6.2} & 0 \end{bmatrix} \tag{3.6}$$

#### Input Matrix

Similarly,  $f_1$  and  $f_2$  from 2.4 and 2.5 may be used to obtain a linearized input matrix. The input matrix general form is given in equation 3.7. Using equations 3.8, this gives 3.9.

$$\mathbf{B} = \begin{bmatrix} \frac{df_1}{du} \\ \frac{df_2}{du} \end{bmatrix} \Big|_{x^*, u^*} \tag{3.7}$$

$$\frac{df_1}{du} = 0 \qquad \qquad \frac{df_2}{du} = \frac{9523.809524}{m \left(x_1 + 6.2\right)^4} \tag{3.8}$$

$$\mathbf{B} = \begin{bmatrix} 0\\ \frac{9523.809524}{m(z_0 + 6.2)^4} \end{bmatrix} \tag{3.9}$$

#### **Output Matrix**

The output matrix is simply given by equation 3.10

$$\mathbf{C} = \begin{bmatrix} 1 & 0 \end{bmatrix} \tag{3.10}$$

## 3.3 Linearized System Model

The final linearized model of the is given by equations 3.11.

$$\dot{\mathbf{x}} - \dot{\mathbf{x}}^* = \begin{bmatrix} 0 & 1 \\ -\frac{39.24}{z_0 + 6.2} & 0 \end{bmatrix} (\mathbf{x} - \mathbf{x}^*) + \begin{bmatrix} 0 \\ \frac{9523.809524}{m(z_0 + 6.2)^4} \end{bmatrix} (u - u^*)$$
(3.11)

## 4

# Open Loop Analysis

## 4.1 Stability

System stability depends on the eigenvalues of matrix **A** from equation 3.6. The eigenvalues are shown in equation 4.1. For simplicity, initial position is assumed to be  $z_0 = 1$ . This yields equation 4.2.

$$\lambda_1 = \frac{65.4}{\sqrt{-109 \, z_0 - 675.8}} \qquad \lambda_2 = -\frac{65.4}{\sqrt{-109 \, z_0 - 675.8}} \tag{4.1}$$

$$\lambda_1 = 0 + 2.33 j$$
  $\lambda_2 = 0 - 2.33 j$  (4.2)

Since the real part of these values is zero, the system is stable in the sense of Laypunov. This result is expected as friction was assumed to be zero.

#### 4.2 Controllability

The controllability matrix for this system is shown in equation 4.3. It can easily be seen that the matrix is full rank. As such, the system is controllable.

$$\mathbf{\Phi_c} = \begin{bmatrix} \mathbf{B} & | & \mathbf{AB} \end{bmatrix} = \begin{bmatrix} 0 & \frac{0.138}{m} \\ \frac{0.138}{m} & 0 \end{bmatrix}$$
(4.3)

## 4.3 Observability

The observability matrix for this system is shown in equation 4.4. It can easily be seen that the matrix is full rank. As such, the system is observable.

$$\mathbf{\Phi_o} = \begin{bmatrix} \mathbf{C} \\ \mathbf{CA} \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \tag{4.4}$$

# References

- [1] E. C. Products, Manual For Model 730 Magnetic Levitation System, 1 Buckskin Court, Bell Canyon, California 91307, 1999.
- [2] P. S. V. Nataraj and M. D. Patil, "Robust Control Design for Nonlinear Magnetic Levitation System using Quantitative Feedback Theory (QFT)," *IEEE*, 2008.