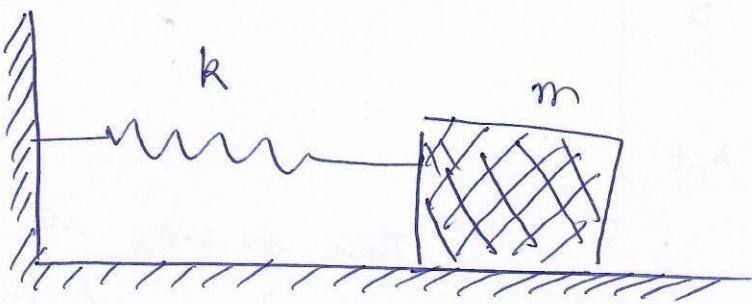


forced spring Mass system

Week 3

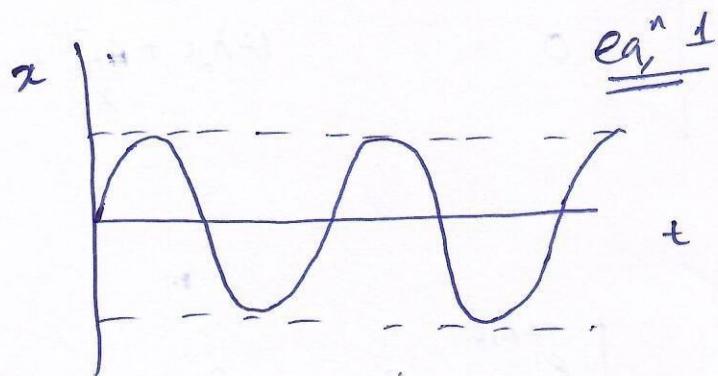
Revision



$$\left\{ \begin{array}{l} m \frac{d^2x}{dt^2} + kx = 0 \quad \text{--- (1)} \\ \text{free undamped system} \end{array} \right.$$

$$\left\{ \begin{array}{l} m \frac{d^2x}{dt^2} + c \frac{dx}{dt} + kx = 0 \quad \text{--- (2)} \\ \text{free vibration with damping} \end{array} \right.$$

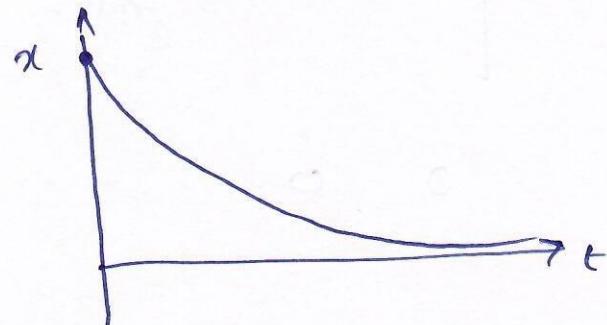
autonomous



eq (2)

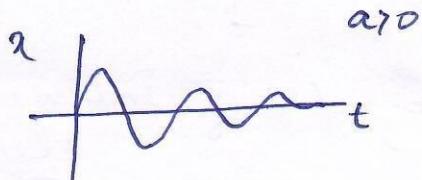
overdamped system

$$\text{form of solution} \rightarrow x(t) = C_1 e^{-\lambda_1 t} + C_2 e^{-\lambda_2 t}$$



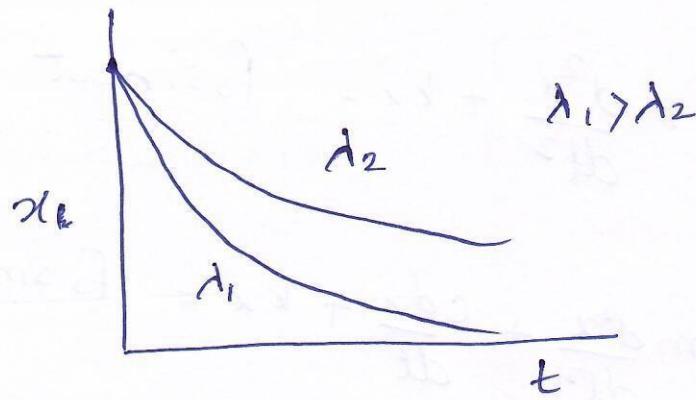
Underdamped system

$$x(t) = e^{-at} (\psi_1 \sin bt + \psi_2 \cos bt)$$



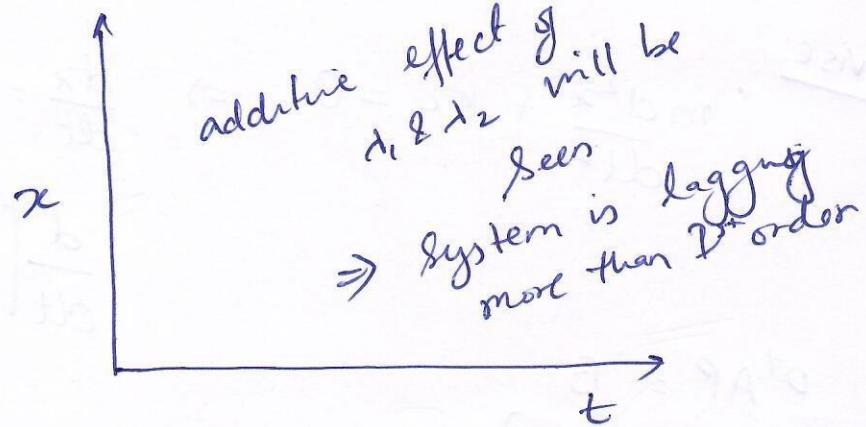
In case of overdamped

In case of
1st order



In case of
second order

critical
damping



$$x(t) = C_1 e^{-\lambda_1 t} + C_2 e^{-\lambda_2 t}$$

$$\frac{dx}{dt} = -\underline{\lambda} x$$

L ↳ eigen values. (λ_1, λ_2)

$$\text{if } \lambda_1 = \lambda_2$$

$$x(t) = \underbrace{C_1 e^{-\lambda_1 t} + C_2 t e^{-\lambda_1 t}}_{\text{critical damping}} \rightarrow \underline{\text{Prove}}$$

for Forced system — non autonomous.

$$m \frac{d^2x}{dt^2} + kx = f_0 \sin \omega t$$

$$m \frac{d^2x}{dt^2} + c \frac{dx}{dt} + kx = f_0 \sin \omega t$$

Revise

$$m \frac{d^2x}{dt^2} + kx = 0 \Rightarrow \frac{dx}{dt} = y; \frac{dy}{dt} = -\frac{k}{m}x$$

$$\frac{d}{dt} \begin{bmatrix} x \\ y \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -\frac{k}{m} & 0 \end{bmatrix} \begin{bmatrix} x \\ y \end{bmatrix}$$

$$\underline{P}^T \underline{A} \underline{P} = \underline{\Lambda}$$

$$\underline{P} = \begin{bmatrix} v_1 & v_2 \end{bmatrix}$$

$$\underline{P}^T \underline{A} \underline{P} = \underline{\Lambda}$$

$$\underline{\Lambda} = \begin{bmatrix} \lambda_1 & 0 \\ 0 & \lambda_2 \end{bmatrix}$$

$$\frac{d\underline{x}}{dt} = \underline{\Lambda} \underline{x}$$

↳ diagonalise \underline{A}

$$\underline{P} = \begin{bmatrix} v_1 & v_2 \end{bmatrix}$$

$$\underline{P} = \begin{bmatrix} i\sqrt{\frac{m}{k}} & -i\sqrt{\frac{m}{k}} \\ 1 & 1 \end{bmatrix}$$

$$\underline{P}^T = \frac{1}{2} \begin{bmatrix} -i\sqrt{\frac{k}{m}} & i\sqrt{\frac{k}{m}} \\ 1 & 1 \end{bmatrix}$$

$$\underline{\Lambda} = \begin{bmatrix} -i\sqrt{\frac{k}{m}} & 0 \\ 0 & i\sqrt{\frac{k}{m}} \end{bmatrix}$$

$$\frac{dx}{dt} = Ax$$

$$\frac{d}{dt}(P^T x) = (P^T A P) P^T x$$

$$\frac{d}{dt}(P^T x) = \underline{\Lambda}(P^T x)$$

$$\frac{d}{dt} \underline{y} = \underline{\Lambda} \underline{y}$$

$$y = P^T x$$

$$\underline{\Lambda} = P^T A P$$

\hookrightarrow

$$\underline{y} = e^{\underline{\Lambda} t} \underline{c}$$

$$\underline{c} = \begin{bmatrix} c_1 \\ c_2 \end{bmatrix} \text{ initial condition}$$

$$\begin{bmatrix} y_1 \\ y_2 \end{bmatrix} =$$

~~good~~

Now, transform y back to x

$$\underline{y} = P^T x \Rightarrow x = P \underline{y}$$

$$\underline{y} = \begin{bmatrix} e^{(i\sqrt{\frac{k}{m}})t} & 0 \\ 0 & e^{(i\sqrt{\frac{k}{m}})t} \end{bmatrix} \begin{bmatrix} c_1 \\ c_2 \end{bmatrix}$$

$$\underline{x} = \begin{bmatrix} i\sqrt{\frac{m}{k}} & -i\sqrt{\frac{m}{k}} \\ 1 & 1 \end{bmatrix} \begin{bmatrix} e^{-i\sqrt{\frac{k}{m}}t} & 0 \\ 0 & e^{i\sqrt{\frac{k}{m}}t} \end{bmatrix} \begin{bmatrix} c_1 \\ c_2 \end{bmatrix}$$

$$\begin{bmatrix} x \\ \frac{dx}{dt} \end{bmatrix} = \underline{x} = \begin{bmatrix} \cdot & \cdot & \cdot & \cdot \\ \cdot & \cdot & \cdot & \cdot \end{bmatrix}$$

So for forced.

$$m \frac{d^2x}{dt^2} + kx = f_0 \sin \omega t$$

$$m \frac{dy}{dt} + kx = f_0 \sin \omega t ; \quad \cancel{\text{by}} \quad \frac{dx}{dt} = y$$

$$\frac{d}{dt} \begin{bmatrix} x \\ y \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -\frac{k}{m} & 0 \end{bmatrix} \begin{bmatrix} x \\ y \end{bmatrix} + \begin{bmatrix} 0 \\ f_0 \sin \omega t \end{bmatrix}$$

$$\stackrel{A}{=} \lambda_1$$

$$\lambda_2$$

$$v_1$$

$$v_2$$

$$P =$$

$$P^+ =$$

$$\Delta =$$

Same as
autonomous.

$$\frac{d\underline{x}}{dt} = \underline{\underline{A}} \underline{x} + \underline{g}(t) \quad \text{--- (1)}$$

$$\frac{d}{dt} (\underline{P}^T \underline{x}) = (\underline{P}^T \underline{A} \underline{P}) \underline{P}^T \underline{x} + \underline{P}^T \underline{g}(t)$$

$$\frac{d}{dt} (\underline{P}^T \underline{x}) = \underline{\underline{N}} (\underline{P}^T \underline{x}) + \underline{P}^T \underline{g}(t)$$

$$\text{let } \underline{P}^T \underline{x} = \underline{y}$$

$$\frac{d}{dt} \underline{y} = \underline{\underline{N}} \underline{y} + \underline{b}$$

$$\underline{P}^T \underline{g}(t) = \underline{b}$$

$$\frac{d}{dt} \underline{y} - \underline{\underline{N}} \underline{y} = \underline{b} \quad \text{--- (2)}$$

$$\text{If } \underline{y} = e^{-\underline{\underline{N}} t}$$

multiply both sides & solve for \underline{y}

$$\text{But } \underline{b} = ?!$$

$$\underline{b} = \underline{P}^T \underline{g}$$

$$\underline{b} = \frac{1}{2} \begin{bmatrix} -i\sqrt{\frac{k}{m}} \\ 1 \end{bmatrix}$$

$$\begin{bmatrix} i\sqrt{\frac{k}{m}} \\ 1 \end{bmatrix} \left| \begin{array}{l} 0 \\ \text{for sinut} \end{array} \right.$$

Put \underline{b} in eq 2

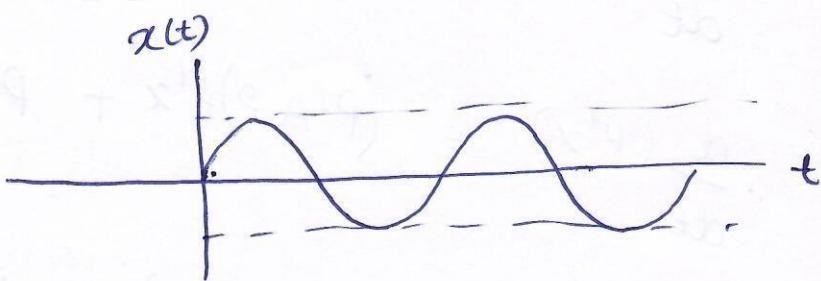
solve for \underline{y}

then solve for \underline{x}

$$\underline{x} = \underline{P} \underline{y}$$

Example

$$1. \frac{d^2x}{dt^2} + x = 0 \Rightarrow x(t) = C_1 \cos t + C_2 \sin t$$



$$2. \frac{d^2x}{dt^2} + x = \sin t \Rightarrow x(t) = C_1 \cos t + C_2 \sin t - \frac{1}{2}t \cos t.$$

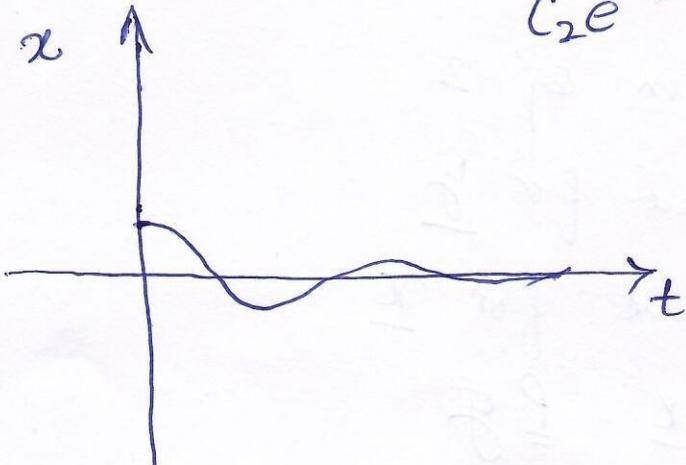
oscillations increase with time because effect of autonomous part is augmented by non-autonomous.

external force is increasing amplitude with time.

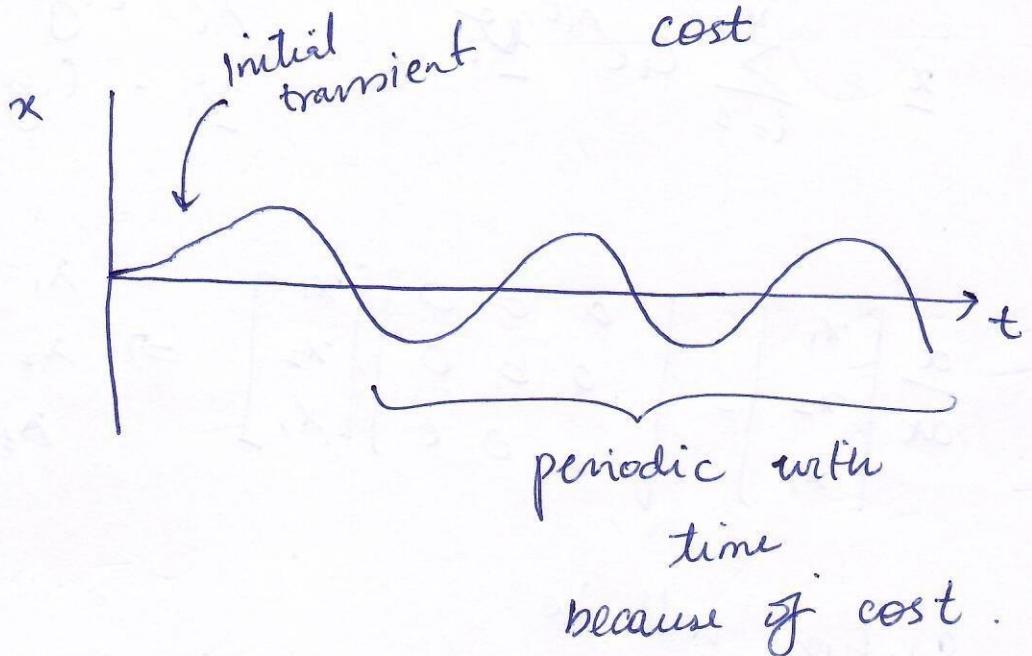


$$3. \frac{d^2x}{dt^2} + \frac{dx}{dt} + x = 0 \Rightarrow x(t) = C_1 e^{-t/2} \sin \sqrt{3/2} t + C_2 e^{-t/2} \cos \sqrt{3/2} t$$

Damping

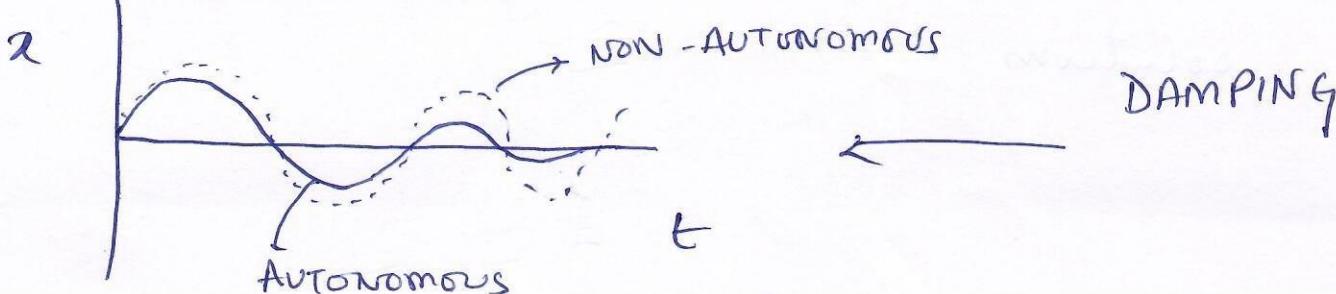
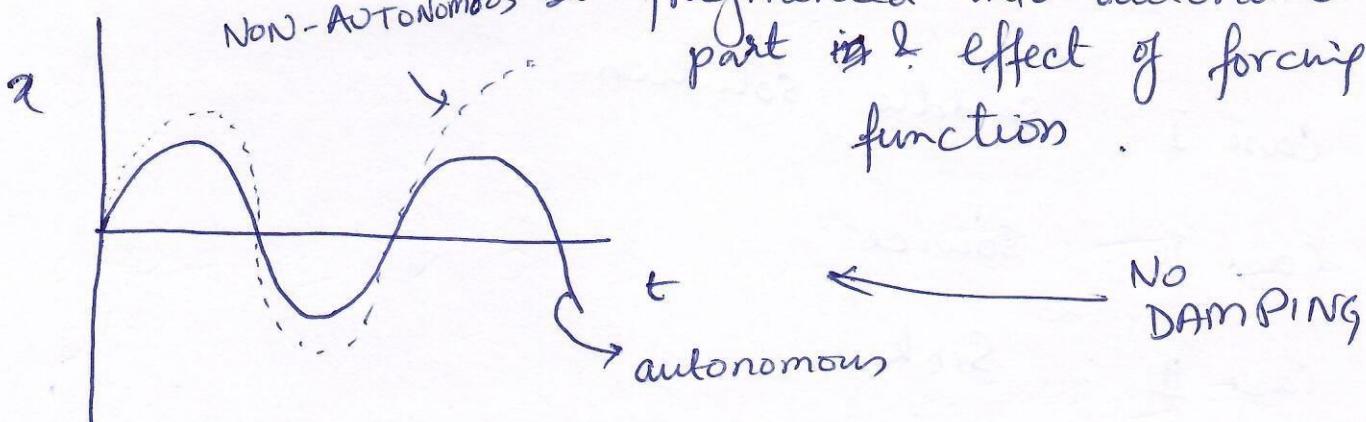


$$4. \frac{d^2x}{dt^2} + \frac{dx}{dt} + x = \sin t \Rightarrow x(t) = C_1 e^{-t/2} \sin \frac{\sqrt{3}}{2} t + C_2 e^{-t/2} \cos \frac{\sqrt{3}}{2} t -$$



effect of first 2 terms will decrease on time

Summary → every non-autonomous system can be fragmented into autonomous part & effect of forcing function.



2 Order system

$$\frac{d\underline{x}}{dt} = \underline{A}\underline{x}$$

$$\begin{cases} \underline{x} = Nx \\ \underline{A} = NxN \end{cases}$$

$$\underline{x} = \sum_{i=1}^N Ge^{\lambda_i t} \underline{x}_i$$

λ_i = Eigen Value

\underline{x}_i = Eigen vectors

$$\frac{d}{dt} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} = \begin{bmatrix} a & 0 & 0 \\ 0 & b & 0 \\ 0 & 0 & c \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} \Rightarrow \begin{array}{ll} \lambda_1 = a & \underline{x}_1 = [1 \ 0 \ 0] \\ \lambda_2 = b & \underline{x}_2 = [0 \ 1 \ 0] \\ \lambda_3 = c & \underline{x}_3 = [0 \ 0 \ 1] \end{array}$$

Solution will be

$$\begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} = C_1 e^{at} \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} + C_2 e^{bt} \begin{bmatrix} 0 \\ 1 \\ 0 \end{bmatrix} + C_3 e^{ct} \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} \quad \textcircled{1}$$

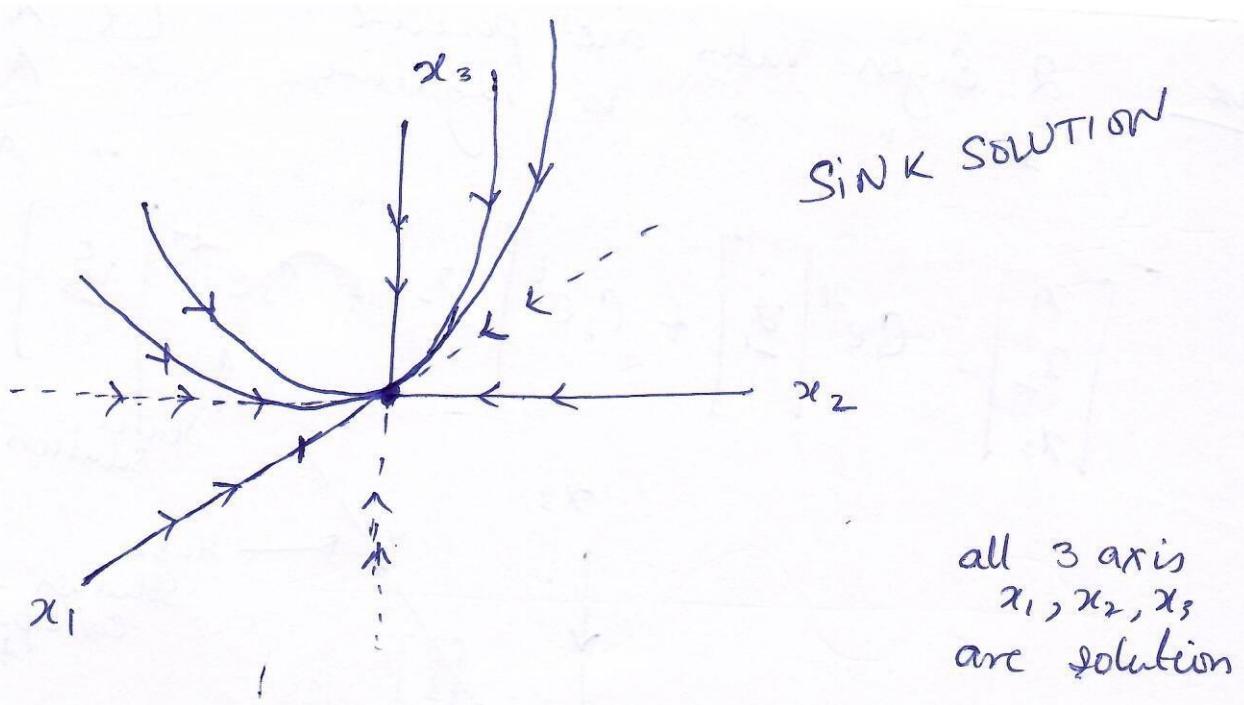
$$a, b, c \in \mathbb{R}$$

Case I : Saddle solution $a>0, b<0$

Case II : Source $a>0, b>0$

Case III : Sink $a<0, b<0$

for eq¹ 1 ; \underline{x}_1 ; \underline{x}_2 & \underline{x}_3 all are solutions



at equilibrium.

$$\begin{bmatrix} x_{1e} = 0 \\ x_{2e} = 0 \\ x_{3e} = 0 \end{bmatrix} - \text{solution}$$

if $a < 0, b < 0, c < 0$ ← Sink solution

$$\lim_{t \rightarrow \infty} C_1 e^{at} v_1 \rightarrow 0 ; \lim_{t \rightarrow \infty} C_2 e^{bt} v_2 \rightarrow 0 ; \lim_{t \rightarrow \infty} C_3 e^{ct} v_3 \rightarrow 0$$

dominant Eigen value will decide direction
which will act as tangent

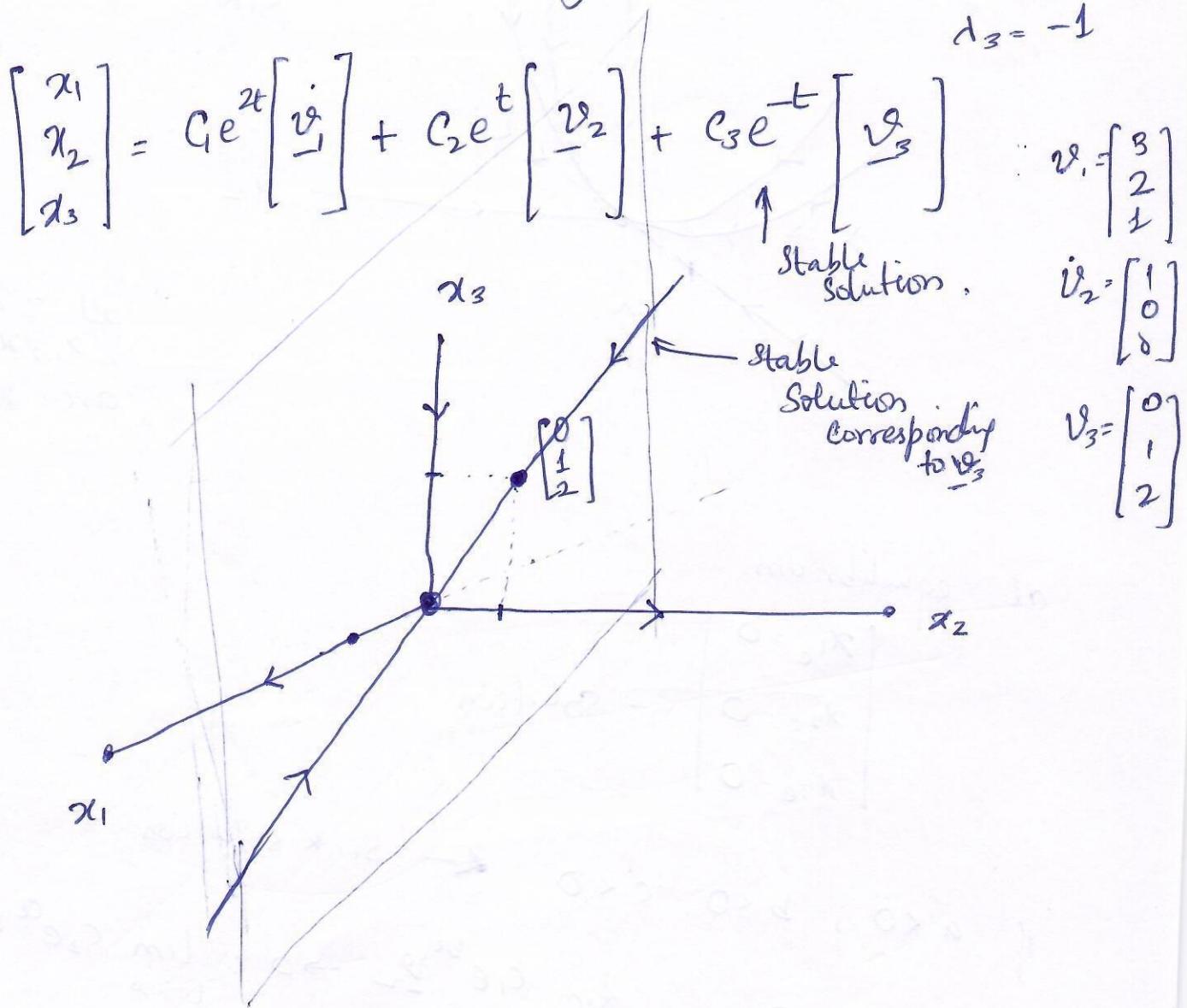
If $a > b > c$ (all are negative)

x_1 will be tangent.

SOURCE SOLUTION WILL BE EXACTLY OPPOSITE

$$a > 0, b > 0, c > 0$$

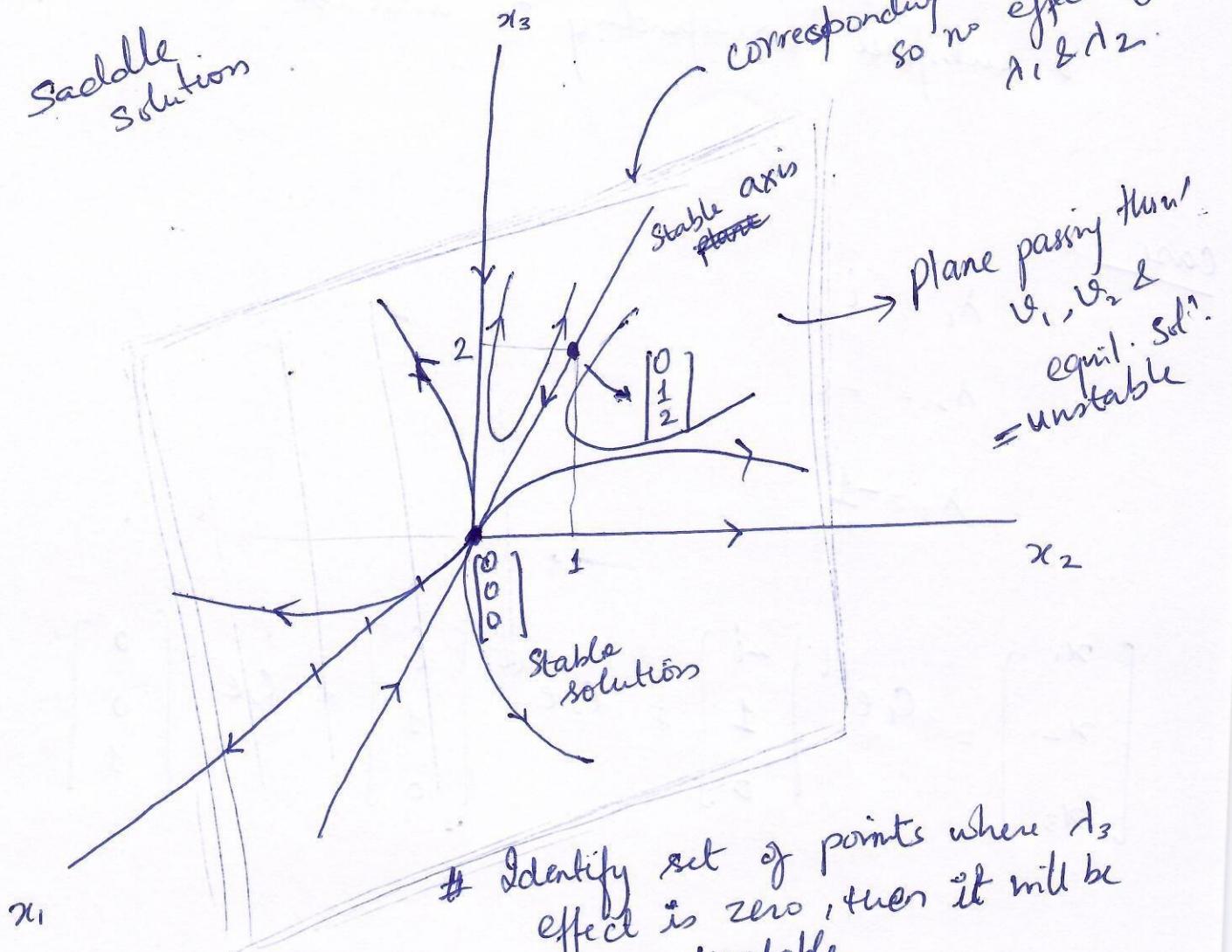
Case 2 Eigen Value are positive
 1 n n 0 negative. Let $\lambda_1 = 2$
 $\lambda_2 = 1$
 $\lambda_3 = -1$



equilibrium solution = $\begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix}$

$$\begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} = C_1 e^{2t} \begin{bmatrix} 3 \\ 2 \\ 1 \end{bmatrix} + C_2 e^t \begin{bmatrix} 1 \\ 0 \\ 0 \end{bmatrix} + C_3 e^{-t} \begin{bmatrix} 0 \\ 1 \\ 2 \end{bmatrix}$$

Saddle solution



corresponding to λ_3
so no effect of
 λ_1 & λ_2

Plane passing thru'
 v_1, v_2 &
equil. sol.
= unstable

Identify set of points where λ_3 effect is zero, then it will be unstable

Plane passing thru v_1, v_2 & equilibrium solution

will be unstable

as $x_3 = 0$; all the points on this plane will cause instability.

Any point on this plane is unstable

So, We have subspace corresponding to stable solution & a subspace corresponding to unstable solution.

case

$$\lambda_1 = i$$

$$\lambda_2 = -i$$

$$\lambda_3 = -1$$

$$\begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} = C_1 e^{it} \begin{bmatrix} -i \\ 1 \\ 0 \end{bmatrix} + C_2 e^{-it} \begin{bmatrix} i \\ 1 \\ 0 \end{bmatrix} + C_3 e^{-t} \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}$$

$$e^{it} = \cos t + i \sin t$$

$$e^{-it} = \cos t - i \sin t$$

$$\text{form of} = (\text{Re}) + i(\text{Im})$$

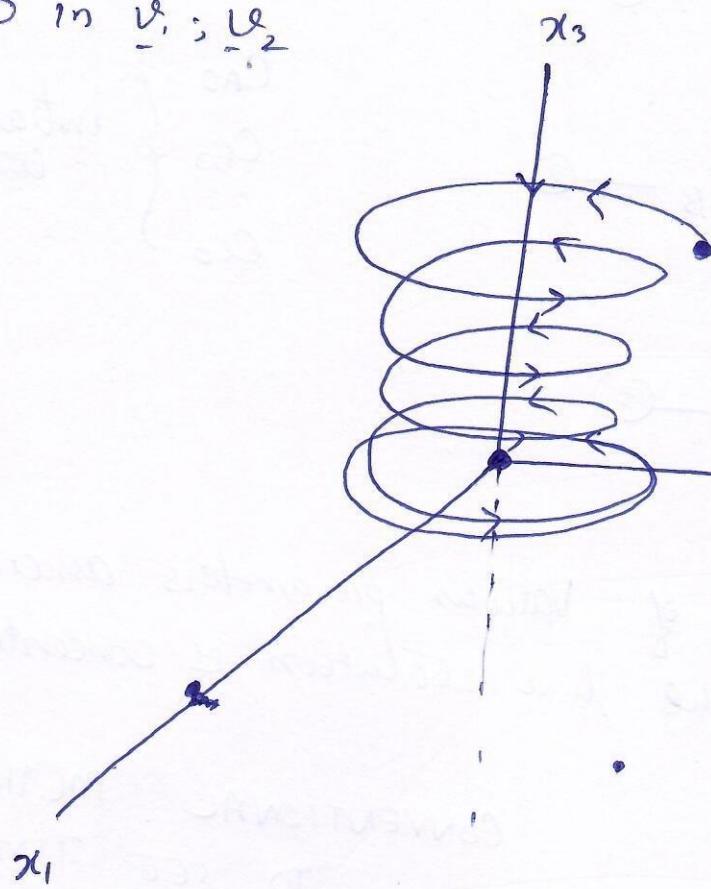
solution.

$$= \text{Re } e^{it} + i \text{Im } e^{it}$$

finally we have

$$\begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} = d_1 \begin{bmatrix} \cos t \\ -\sin t \\ 0 \end{bmatrix} + d_2 \begin{bmatrix} \sin t \\ \cos t \\ 0 \end{bmatrix} + d_3 e^{-t} \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix}$$

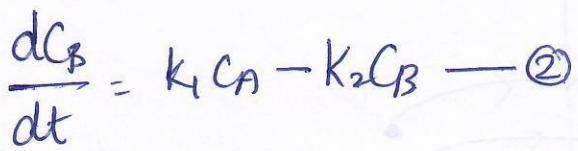
as $x_3 = 0$ in \underline{V}_1 ; \underline{V}_2



Spiral solution moving towards equilibrium solution
on reaching $x_3 = 0$ it will become 2D & become circular solution.

~~A~~ generalised Eigen vector \leftarrow Read about it.

Analysis of complex reaction systems



C_{A0}
 C_{B0}
 C_{C0}

} initial conditions



Q. Analyse the effects of Various parameters associated with system on the time evolution of concentrations of A, B, C

CONVENTIONAL METHOD

$$\text{eq } (1) \quad C_A = C_{A0} e^{-kt}$$

TO SEE TIME
HAP EVOLUTION

eq' (2).

$$\frac{dC_B}{dt} + k_2 C_B = k_1 C_{A0} e^{-kt}$$

$$\hookrightarrow \frac{dx}{dt} + f(x) = g(t)$$

$$e^{k_2 t} \frac{dC_B}{dt} + k_2 e^{k_2 t} C_B = k_1 C_{A0} e^{-kt} \cdot e^{k_2 t}$$

$$\frac{d}{dt} (R^{k_2 t} C_B) = k_1 C_{A0} e^{(k_2 - k_1)t}$$

$$e^{k_2 t} C_B = \left(\frac{k_1}{k_2 - k_1} \right) C_{A0} e^{(k_2 - k_1)t} + C_1$$

$$\text{at } t=0 ; C_B = C_{B0}$$

$$C_{B0} = \left(\frac{k_1}{k_2 - k_1} \right) C_{A0} + C_1$$

$$C_1 = C_{B0} - \left(\frac{k_1}{k_2 - k_1} \right) C_{A0}$$

$$e^{k_2 t} C_B = \left(\frac{k_1}{k_2 - k_1} \right) C_{A0} e^{(k_2 - k_1)t} + C_{B0} - \left(\frac{k_1}{k_2 - k_1} \right) C_{A0}$$

$$C_B = \left(\frac{k_1}{k_2 - k_1} \right) C_{A0} e^{k_2 t} (e^{(k_2 - k_1)t} - 1) + e^{-k_2 t} C_{B0}$$

$$C_B = \left(\frac{k_1}{k_2 - k_1} \right) C_{A0} (e^{-k_1 t} - e^{-k_2 t}) + C_{B0} e^{-k_2 t}$$

eqn ③ $\frac{dC_c}{dt} = k_2 C_B$

$$\frac{dC_c}{dt} = \left(\frac{k_2 k_1}{k_2 - k_1} \right) C_{A0} (e^{-k_1 t} - e^{-k_2 t}) + k_2 C_{B0} e^{-k_2 t}$$

$$C_c = \left(\frac{k_1 k_2}{k_2 - k_1} \right) C_{AO} \left(\frac{1}{k_1} \right) e^{-k_1 t} - \left(\frac{k_1 k_2}{k_2 + k_1} \right) C_{AO} \left(\frac{1}{k_2} \right) e^{-k_2 t} - C_{B_0} e^{-k_2 t} + C_2$$

$$C_c = \left(\frac{-k_2}{k_2 + k_1} \right) C_{AO} e^{k_1 t} + \left(\frac{k_1}{k_2 - k_1} \right) C_{AO} e^{-k_2 t} - C_{B_0} e^{-k_2 t} + C_2$$

$$\text{at } t=0 \quad C_c = C_{CO}$$

$$C_{CO} = \left(\frac{-k_2}{k_2 - k_1} \right) C_{AO} + \left(\frac{k_1}{k_2 - k_1} \right) C_{AO} - C_{B_0} + C_2$$

$$C_2 = C_{AO} + C_{B_0} + C_{CO}$$

$$C_c = \left(\frac{-k_2}{k_2 - k_1} \right) C_{AO} e^{-k_1 t} + \left(\frac{k_1}{k_2 - k_1} \right) C_{AO} e^{-k_2 t} - C_{B_0} e^{-k_2 t} + C_{AO} + C_{B_0} + C_{CO}$$

~~C_A~~
then we see how C_A, C_B & C_c are changing with time.

Matrix Solution

$$\frac{dC_A}{dt} = k_1 C_A$$

$$\frac{dC_B}{dt} = k_2 C_A - k_3 C_B$$

$$\frac{dC_C}{dt} = k_3 C_B$$

ORDER - 3

DYNAMIC VARIABLE \rightarrow

$$\begin{bmatrix} C_A \\ C_B \\ C_C \end{bmatrix}$$

linear system

Autonomous eqn

$$\frac{d}{dt} \begin{bmatrix} C_A \\ C_B \\ C_C \end{bmatrix} = \begin{bmatrix} -k_1 & 0 & 0 \\ k_1 & k_2 & 0 \\ 0 & k_3 & 0 \end{bmatrix} \begin{bmatrix} C_A \\ C_B \\ C_C \end{bmatrix} \Rightarrow \lambda_1 = 0 \quad \underline{v}_1 = \begin{bmatrix} 0 & 0 & 1 \end{bmatrix}'$$

$$\lambda_2 = -k_1 \quad \underline{v}_2 = \begin{bmatrix} k_1 - k_2 \\ k_1 \\ -\frac{k_1}{k_2} \end{bmatrix}$$

$$\lambda_3 = -k_2 \quad \underline{v}_3 = \begin{bmatrix} 0 \\ -1 \\ 1 \end{bmatrix}$$

$$\frac{d\underline{x}}{dt} = \underline{\underline{A}} \underline{x}$$

at equilibrium

$$C_{Ae} = 0$$

$$C_{Be} = 0$$

$$C_{Ce} = ? \quad \text{let it be } \alpha$$

$$\alpha = C_{Ae} + C_{Be} + C_{Ce}$$

Revise

- E. Value are purely imaginary = oscillatory behaviour
- complex no. with real part > 0 = oscillatory behaviour with increasing amplitude
- complex no. with real part < 0 = oscillatory behaviour with decreasing amplitude
- Real eigen values > 0 = unstable
- < 0 = stable system

Some E. Value > 0 & some E. Value < 0 = saddle solution

Case 1. $k_1 > 0$; $k_2 \neq 0 \Rightarrow \lambda_2 < 0$; $\lambda_3 < 0$

↳ Stable solution.

$\boxed{\lambda_1 = 0}$ ← How will this impact the system?

$$\begin{bmatrix} C_A \\ C_B \\ C_C \end{bmatrix} = C_1 \begin{bmatrix} 0 \\ 0 \\ 1 \end{bmatrix} + C_2 e^{kt} \begin{bmatrix} \frac{k_1 - k_2}{k_2} \\ -\frac{k_1}{k_2} \\ 1 \end{bmatrix} + C_3 e^{-k_2 t} \begin{bmatrix} 0 \\ -1 \\ 1 \end{bmatrix}$$

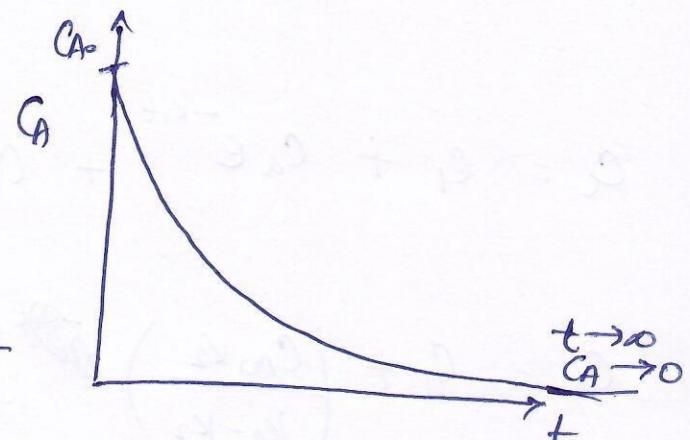
$$c_1 C_A = C_2 \left(\frac{k_1 - k_2}{k_2} \right) e^{-k_1 t} \quad (\text{at } t=0 \quad C_A = C_{A0})$$

$$C_{A0} = C_2 \left(\frac{k_1 - k_2}{k_2} \right)$$

$$C_2 = \left(\frac{C_{A0} k_2}{k_1 - k_2} \right)$$

$$C_A = \left(\frac{k_2 C_{A0}}{k_1 - k_2} \right) \left(\frac{k_1 - k_2}{k_2} \right) e^{-k_1 t}$$

$$C_A = C_{A0} e^{-k_1 t}$$



$$C_B = C_2 e^{-k_1 t} \left(\frac{-k_1}{k_2} \right) + C_3 e^{-k_2 t} \quad \text{at } t=0 \quad C_B = C_{B0}$$

$$C_{B0} = C_2 \left(\frac{-k_1}{k_2} \right) - C_3$$

$$C_3 = C_2 \left(\frac{-k_1}{k_2} \right) - C_{B0}$$

$$C_3 = \left(\frac{C_{A0} k_2}{k_1 - k_2} \right) \left(\frac{-k_1}{k_2} \right) - C_{B0}$$

$$C_B = \left(\frac{C_{A0} k_2}{k_1 - k_2} \right) e^{-k_1 t} \left(\frac{-k_1}{k_2} \right) - \left(\frac{C_{A0} k_2}{k_1 - k_2} \right) \left(\frac{-k_1}{k_2} \right) e^{-k_2 t} + C_{B0} e^{-k_2 t}$$

$$C_B = \left(\frac{C_{A0} k_2}{k_1 - k_2} \right) \left(\frac{-k_1}{k_2} \right) \left[e^{-k_1 t} - e^{-k_2 t} \right] + C_{B0} e^{-k_2 t}$$

$$C_B = \left(\frac{k_1 C_{A0}}{k_1 - k_2} \right) [e^{-k_1 t} - e^{-k_2 t}] + C_{B0} e^{-k_2 t}$$

$$C_C = C_1 + C_a e^{-k_1 t} + C_b e^{-k_2 t} \quad \text{at } t=0 \\ C_C = C_{C0}$$

$$C_{C0} = C_1 + \left(\frac{C_{A0} k_2}{k_1 - k_2} \right) e^{-k_1 t} + \left(\frac{-C_{A0} k_1}{k_1 - k_2} \right) e^{-k_2 t} - C_{B0} e^{-k_2 t}$$

$$C_1 = C_0 - \left(\frac{C_{A0} k_2}{k_1 - k_2} \right) e^{-k_1 t} + \left(\frac{C_{A0} k_1}{k_1 - k_2} \right) e^{-k_2 t} + C_{B0} e^{-k_2 t}$$

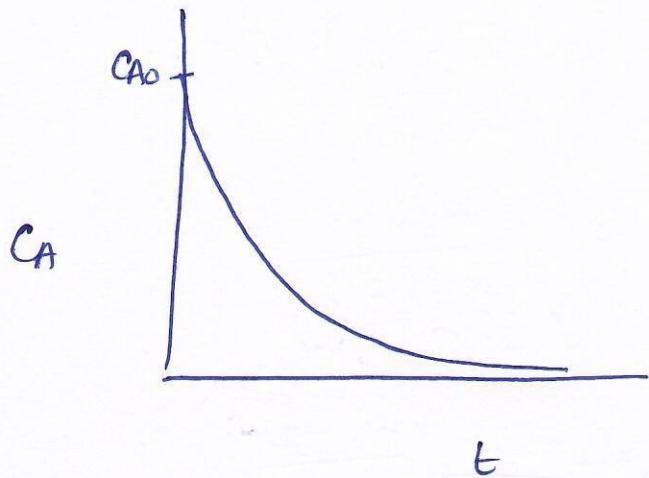
$$C_1 = C_0 - \left(\frac{C_{A0}}{k_1 - k_2} \right) (k_2 e^{-k_1 t} + k_1 e^{-k_2 t}) + C_{B0} e^{-k_2 t}$$

$$C_1 = C_{A0} + C_{B0} + C_{C0}$$

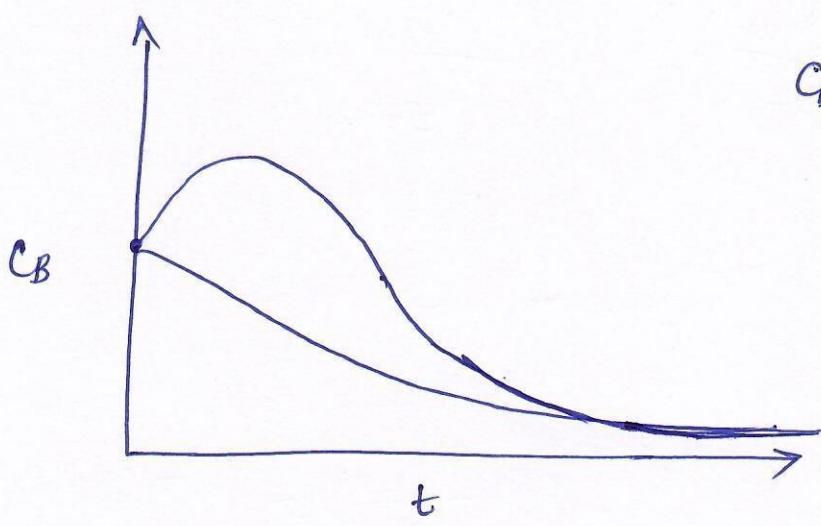
~~C_A=C_B~~ -

$$C_C = C_{A0} + C_{B0} + C_{C0} + \left(\frac{C_{A0} k_2}{k_1 - k_2} \right) e^{-k_1 t} + \left(\frac{C_{A0} k_1}{k_2 - k_1} \right) e^{-k_2 t} - C_{B0} e^{-k_2 t}$$

finally, we have



$$C_A = C_{A0} e^{-k_A t}$$

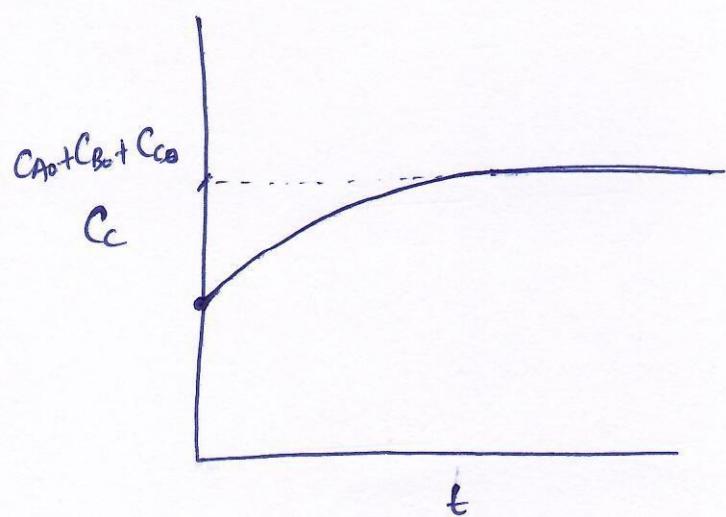


$$C_B = () [e^{-\nu e} - e^{-\nu e}] + () e^{-\nu e}$$

time $\rightarrow \infty$

$$C_B \rightarrow 0$$

depending on initial condition we can have any trajectory



$$C_C = [\text{const}]_0 + () e^{-\nu e} + () e^{-\nu e} + () e^{-\nu e}$$

so as $t \rightarrow \infty$

$$e^{-\nu e} \rightarrow 0$$

so effect of exp. part diminishes.

so at $t \rightarrow \infty$

$$C_C \rightarrow \text{const}$$