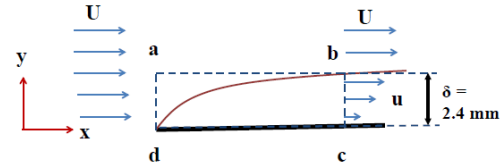


1. Air at standard conditions is flowing over a thin flat plate which is 1 m long and 0.3 m wide. The flow is uniform at the leading edge of the plate. The velocity profile in the boundary layer is assumed to be linear and  $U = 30$  m/s. Assume that the flow conditions are independent of  $z$  and treat the flow as two dimensional. Using the control volume abcd ( $dc = 1$  m), compute the mass flow rate across surface ab. The boundary layer thickness at the end of the plate (point c in the figure) is  $\delta = 2.4$  mm. Determine the magnitude and direction of the x-component of the force required to hold the plate stationary ( $\rho_{\text{air}} = 1.23$  kg/m<sup>3</sup>). **4+6=10**

1. At bc,  $\frac{u}{U} = \frac{y}{\delta} = \eta$ ,  $U = 30$  m/s

Basic equations:  $0 = \frac{\partial}{\partial t} \int_{CV} \rho dV + \int_{CS} \rho \vec{V} \cdot d\vec{A}$



$$F_{SX} + F_{BX} = \frac{\partial}{\partial t} \int_{CV} u \rho dV + \int_{CS} u \rho \vec{V} \cdot d\vec{A}$$

$$\therefore 0 = \underbrace{\{-\rho U b \delta\}}_{\dot{m}_{ad}} + \int_0^\delta \rho U b dy + \dot{m}_{ab}, \quad \text{and} \quad \int_0^\delta \rho U b dy = \rho U b \delta \int_0^1 \eta d\eta = \frac{1}{2} \rho U b \delta = \dot{m}_{bc}$$

$$\text{Or } \dot{m}_{ab} = \frac{1}{2} \rho U b \delta = \frac{1}{2} \times 1.23 \frac{\text{kg}}{\text{m}^3} \times 30 \frac{\text{m}}{\text{s}} \times 0.3 \text{ m} \times 0.0024 \text{ m} = 0.0133 \frac{\text{kg}}{\text{s}}$$

From momentum equation

$$R_x = u_{da} \{-\rho U b \delta\} + u_{ab} \dot{m}_{ab} + \int_0^\delta \rho u^2 b dy$$

$$\text{Now } \int_0^\delta \rho u^2 b dy = \rho U^2 b \delta \int_0^1 \eta^2 d\eta = \frac{1}{3} \rho U^2 b \delta$$

And  $u_{da} = U, u_{ab} = U$  (all of ab is outside BL)

$$R_x = -\rho U^2 b \delta + \frac{1}{2} \rho U^2 b \delta + \frac{1}{3} \rho U^2 b \delta = -\frac{1}{6} \rho U^2 b \delta$$

$$R_x = -\frac{1}{6} \times 1.23 \frac{\text{kg}}{\text{m}^3} \times (30)^2 \frac{\text{m}^2}{\text{s}^2} \times 0.3 \text{ m} \times 0.0024 \text{ m} = -0.133 \text{ N}$$

This force must be applied to the CV by the plate. Thus, to hold the plate,

$$F_x = R_x = -0.133 \text{ N}$$

2. The presence of CO<sub>2</sub> in solution is essential to the growth of aquatic plant life, with CO<sub>2</sub> used as a reactant in the photosynthesis. Consider a stagnant body of water in which the concentration of CO<sub>2</sub> ( $\rho_A$ ) is everywhere zero. At time  $t = 0$ , the water is exposed to a source of CO<sub>2</sub>, which maintains the surface ( $x = 0$ ) concentration at a fixed value  $\rho_{A0}$ . For time  $t > 0$ , CO<sub>2</sub> will begin to accumulate at the water-air surface and start diffusing in the water with the simultaneous utilization in the photosynthesis process, the rate of which can be expressed as the product of a homogeneous reaction rate constant,  $k_1$  and the local CO<sub>2</sub> concentration  $\rho_A(x, t)$ .
- (a) Modify the species balance equation (expressed in terms of CO<sub>2</sub> mass concentration  $\rho_A$ ) by cancelling the terms (with one-word reason) that are not relevant. What does each term in the equation represent physically?
- b) For this problem, assume the case of a “deep” body of water and write appropriate boundary conditions that could be used to obtain the solution. Also assume negligible CO<sub>2</sub> consumption in the photosynthesis process and modify the governing equation accordingly.
- c) For the condition of Part (b), evaluate the concentration distribution of CO<sub>2</sub> in the stagnant body of the water. **3+3+3=9**

**ASSUMPTIONS:** (1) One-dimensional diffusion in  $x$ , (2) Constant properties, including total density  $\rho$ , (3) Water is stagnant.

The species balance equation in mass concentration terms is

$$\left( \frac{\partial \rho_A}{\partial t} + v_x \frac{\partial \rho_A}{\partial x} + v_y \frac{\partial \rho_A}{\partial y} + v_z \frac{\partial \rho_A}{\partial z} \right) = D_{AB} \left[ \frac{\partial^2 \rho_A}{\partial x^2} + \frac{\partial^2 \rho_A}{\partial y^2} + \frac{\partial^2 \rho_A}{\partial z^2} \right] + R_A$$

For the current problem –

$$D_{AB} \frac{\partial^2 \rho_A}{\partial x^2} - k_1 \rho_A = \frac{\partial \rho_A}{\partial t}$$

The first term on the LHS represents *net* transport of CO<sub>2</sub> into a differential control volume by diffusion.

The second term represents the rate of CO<sub>2</sub> consumption due to chemical reactions.

The term on the right-hand side represents the rate of increase of CO<sub>2</sub> storage within the control volume.

(b) For a deep body of water, appropriate boundary conditions are

$$\begin{aligned} \rho_A(0, t) &= \rho_{A0} \\ \rho_A(\infty, t) &= 0 \end{aligned}$$

and, with negligible chemical reactions, the species diffusion equation reduces to

$$\frac{\partial^2 \rho_A}{\partial x^2} = \frac{1}{D_{AB}} \frac{\partial \rho_A}{\partial t}$$

(c) With an initial condition,  $\rho_A(x, 0) = \rho_{A,i} = 0$ , the problem is analogous to that involving heat transfer in a semi-infinite medium with constant surface temperature. By analogy, the species concentration is then

$$\begin{aligned} \frac{\rho_A(x, t) - \rho_{A,0}}{-\rho_{A,0}} &= \operatorname{erf} \left( \frac{x}{2(D_{AB}t)^{1/2}} \right) \\ \rho_A(x, t) &= \rho_{A,0} \operatorname{erfc} \left( \frac{x}{2(D_{AB}t)^{1/2}} \right) \end{aligned}$$

3. Dry air flows at  $T_\infty = 300\text{K}$  over trays filled with water with a velocity of  $15\text{ m/s}$ . The water surface temperature is maintained at  $330\text{ K}$  by radiant heaters. Find (a) Evaporative flux ( $\text{kg/s.m}^2$ ) at a distance of  $1\text{ m}$  from the leading edge, and (b) The heat flux at this distance required to maintain water temperature at  $330\text{K}$ . The following properties are known: Air: kinematic viscosity  $= 17.40 \times 10^{-6}\text{ m}^2/\text{s}$ ,  $k = 0.0274\text{ W/m.K}$ ,  $\text{Pr} = 0.705$ , Water vapour-air:  $D_{AB} = 0.28 \times 10^{-4}\text{ m}^2/\text{s}$ ,  $\text{Sc} = 0.616$ , Saturated water vapour  $\rho_{A, \text{sat}} = 0.1134\text{ kg/m}^3$ ,  $h_{fg} = 2366\text{ KJ/kg}$  **3+5=8**

(a) The evaporative flux of water vapor (A) is

$$\dot{m}_{A,x} = h_{m,x}(\rho_{A,s} - \rho_{A,\infty}) = h_{m,x}(\rho_{A,\text{sat}}(T_s) - \phi_\infty \rho_{A,\text{sat}}(T_\infty))$$

Evaluate  $\text{Re}_x$  to determine the nature of the flow and then select the proper correlation.

$$\text{Re}_x = \frac{u_\infty x}{\nu} = \frac{15 \frac{\text{m}}{\text{s}} \times 1\text{m}}{17.40 \times 10^{-6} \frac{\text{m}^2}{\text{s}}} = 8.621 \times 10^5$$

Hence the flow is turbulent, and invoking the heat-mass analogy

$$\text{Sh}_x = \frac{h_{m,x}}{D_{AB}} = 0.0296 \text{Re}_x^{\frac{4}{5}} \text{Sc}^{\frac{1}{3}}$$

$$h_m = \frac{0.28 \times 10^{-4} \frac{\text{m}^2}{\text{s}}}{1\text{m}} \times 0.0296 (8.621 \times 10^5)^{\frac{4}{5}} (0.616)^{\frac{1}{3}} = 3.952 \times 10^{-2} \frac{\text{m}}{\text{s}}$$

Hence the evaporative flux at  $x = 1\text{ m}$  is

$$\dot{m}_{A,x} = 3.952 \times 10^{-2} \frac{\text{m}}{\text{s}} \left( 0.1134 \frac{\text{kg}}{\text{m}^3} - 0 \right) = 4.48 \times 10^{-3} \frac{\text{kg}}{\text{s.m}^2}$$

(b) From an energy balance on the differential element at  $x = 1\text{ m}$ ,

$$q_{\text{rad}}'' = q_{\text{conv}}'' + q_{\text{evap}}'' = h_x (T_s - T_\infty) + \dot{m}_{A,x} h_{fg}$$

To estimate  $h_x$ , invoke the heat-mass analogy using the correlation

$$\frac{\text{Nu}_x}{\text{Sh}_x} = \left( \frac{\text{Pr}}{\text{Sc}} \right)^{\frac{1}{3}} \quad \text{or} \quad h_x = h_{m,x} \frac{k}{D_{AB}} \left( \frac{\text{Pr}}{\text{Sc}} \right)^{\frac{1}{3}};$$

$$h_x = 3.952 \times 10^{-2} \frac{\text{kg}}{\text{s.m}^2} \frac{0.0274 \frac{\text{W}}{\text{m.K}}}{0.28 \times 10^{-4} \frac{\text{m}^2}{\text{s}}} \left( \frac{0.705}{0.616} \right)^{\frac{1}{3}} = 40.45 \frac{\text{W}}{\text{m}^2 \text{K}}$$

Hence, the radiant flux is

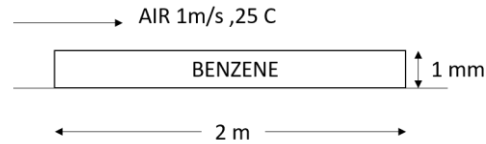
$$q_{\text{rad}}'' = 40.45 \frac{\text{W}}{\text{m}^2 \text{K}} (330 - 300) \text{K} + 4.48 \times 10^{-3} \frac{\text{kg}}{\text{s.m}^2} \times 2366 \times 10^3 \frac{\text{J}}{\text{Kg}}$$

$$q_{\text{rad}}'' = 1214 \frac{\text{W}}{\text{m}^2} + 10600 \frac{\text{W}}{\text{m}^2} = 11813 \frac{\text{W}}{\text{m}^2}$$

4. Benzene, a harmful chemical, has been spilled on the laboratory floor and has spread to a length of 2m. If a film 1 mm depth is formed, how long will it take for the Benzene to completely evaporate? Ventilation in the laboratory provides for airflow parallel to the surface at 1 m/s, and the benzene and air are both at 25 C. The mass densities of Benzene in the saturated vapor and liquid states are known to be 0.417 and 900 kg/m<sup>3</sup>, respectively. For Air: kinematic viscosity = 15.7x10<sup>-6</sup> m<sup>2</sup>/s; D<sub>AB</sub> = 0.88x10<sup>-5</sup> m<sup>2</sup>/s, Sc = 1.78. 9

$$3. \quad \rho_{sat} = 0.417 \frac{kg}{m^3}$$

$$\rho_{liq} = 900 \frac{kg}{m^3}$$



Calculate transition length  $x_{tr}$  (Lam- Turb)

$$\text{Taking } Re_{tr} = 5 \times 10^5 \quad x_{tr} = 7.85m$$

Thus, the film will be under laminar flow.

Mass balance IN-OUT = ACCUM

$$-h_m A \Delta C \Delta t - A \Delta H \rho = 0$$

$$-\frac{d}{dt}(\rho A H) = \dot{m} = \bar{h}_m A \Delta C$$

$$\frac{dH}{dt} = -\frac{\bar{h}_m \Delta C}{\rho}, \text{ at } t = 0, H = h = 1mm$$

$$H = \frac{\bar{h}_m t \Delta C}{\rho_L} \Rightarrow t = \frac{H \rho_L}{\bar{h}_m \Delta C}$$

Evaluate  $\bar{h}_m$

$$Re_L = \frac{2 \times 1}{15.76 \times 10^{-6}} = 126903$$

$$Sh_L = 0.664 Re_L^{\frac{1}{2}} Sc^{\frac{1}{3}}$$

$$\bar{h}_m = 1.26 \times 10^{-3}$$

$$t = \frac{H \rho_L}{\bar{h}_m \Delta C} = \frac{1 \times 10^{-3} \times 900}{1.26 \times 10^{-3} \times 0.417} s \quad t = 1711s$$