

# RBE 501 Week 6: Physics by EL Assignment

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**Abstract**—This document provides an in-depth solution for the Week 6 Physics by Euler-Lagrange problems in RBE 501.

**Index Terms**—physics, euler-lagrange equations

## I. INTRODUCTION

For Week 6, we are given two problems. The first problem shows us a mass  $m$  that is free to move on the surface of a frictionless table. Mass  $m$  is attached to another mass  $M$  via a string that goes through a hole in the table. We can assume the string has no spring-like properties, and is therefore always taut. The representation of this system is shown in Figure 1. Our first objective in this problem is to solve for the Euler-Lagrange equations of this system with respect to  $r$  and  $\theta$ . Our second and third objectives here are to discuss the behaviors of  $r$  and  $\theta$  in the EL equations. The fourth objective of the first problem is to give the conditions under which  $\dot{r}$  and  $\dot{\theta}$  cause a circular motion for mass  $m$ .

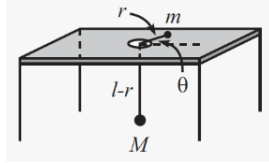


Fig. 1. Given figure for Problem 1

The second problem shows a mass  $m$  that is held at rest on ramp of mass  $M$ . The ramp has inclination  $\theta$ , as shown in Figure 2. The surface of the ramp is frictionless, and the surface upon which the ramp rests is also frictionless. Our objective is to find the acceleration of the ramp caused by the release of mass  $m$ , and we must use Euler-Lagrange approach for this.

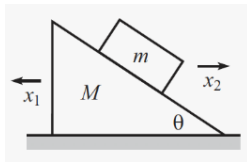


Fig. 2. Given figure for Problem 2

## II. MATERIALS AND METHODS

### A. Approach for Problem I

Our first step toward solving Problem I is to find the overall kinetic energy ( $T$ ) and overall potential energy ( $U$ ) of the system. We then find the Lagrangian of the system as  $L \equiv T - U$ . We can then use the Euler-Lagrangian equation,

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{r}} \right) = \frac{\partial L}{\partial r} \quad (1)$$

and

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{\theta}} \right) = \frac{\partial L}{\partial \theta} \quad (2)$$

to discuss the behaviors of  $r$  and  $\theta$ .

1) *Find overall kinetic energy of the system ( $T$ ):* In Fig. 1, we consider vector  $\mathbf{r}$ . Say the  $x$ - $y$  plane is the surface of the table. Since we consider an angle  $\theta$  as shown, we can say,

$$\mathbf{r} = \begin{pmatrix} r \cos \theta \\ r \sin \theta \end{pmatrix}$$

Where the magnitude of  $r$  is dependent on theta. Differentiating with respect to  $\theta$  to get velocity,

$$\dot{\mathbf{r}} = \begin{pmatrix} \dot{r} \cos \theta - r \dot{\theta} \sin \theta \\ \dot{r} \sin \theta + r \dot{\theta} \cos \theta \end{pmatrix}$$

To find the kinetic energy, we need the square of this velocity, which is simply the dot product of the vector with itself.

$$\begin{aligned} \dot{\mathbf{r}}^2 &= \dot{\mathbf{r}} \cdot \dot{\mathbf{r}} = (\dot{r} \cos \theta - r \dot{\theta} \sin \theta)^2 + (\dot{r} \sin \theta + r \dot{\theta} \cos \theta)^2 \\ &= \dot{r}^2 \cos^2 \theta + r^2 \dot{\theta}^2 \sin^2 \theta + 2r\dot{r}\dot{\theta} \cos \theta \sin \theta + \\ &\quad \dot{r}^2 \sin^2 \theta + r^2 \dot{\theta}^2 \cos^2 \theta - 2r\dot{r}\dot{\theta} \cos \theta \sin \theta \\ &= \dot{r}^2 (\cos^2 \theta + \sin^2 \theta) + r^2 \dot{\theta}^2 (\cos^2 \theta + \sin^2 \theta) \end{aligned}$$

Hence,

$$\dot{\mathbf{r}}^2 = \dot{r}^2 + r^2 \dot{\theta}^2$$

Therefore the kinetic energy of the mass  $m$  moving on the table is given as,

$$T_m = \frac{1}{2} m (\dot{r}^2 + r^2 \dot{\theta}^2) \quad (3)$$

Also, the length of the string that suspends mass  $M$  is  $l - r$ , where total length of the string is  $l$ . The suspended part of the string is also along the axes perpendicular to the table, which is the  $z$  axis. Therefore, the kinetic energy of the mass  $M$  moving perpendicular to the table is given as

$$T_M = \frac{1}{2} M \left( \frac{d}{dt}(l - r)\hat{\mathbf{z}} \right)^2 = \frac{1}{2} M \dot{r}^2 \quad (4)$$

So, using Equations 3 and 4, the total kinetic energy of the system ( $T$ ) is,

$$\begin{aligned} T &= T_m + T_M \\ &= \frac{1}{2} m (\dot{r}^2 + r^2 \dot{\theta}^2) + \frac{1}{2} M \dot{r}^2 \end{aligned}$$

Rearranging,

$$T = \frac{1}{2} \dot{r}^2 (m + M) + \frac{1}{2} m r^2 \dot{\theta}^2 \quad (5)$$

2) *Find overall potential energy of system:* The potential energy is also a sum of the potential energies of the masses  $m$  and  $M$  in the system. The only source of potential energy in this system is gravity caused by the earth. Since we are free to define the reference point from where we can count the height from earth, let us define the table-top in Figure 1 as height  $h = 0$ . Also,  $g$  is the constant acceleration due to gravity.

This means,

$$U_m = mgh = mg(0) = 0$$

and,

$$U_M = Mg(-(l - r)) = -Mg(r - l)$$

So, the overall potential energy of the system is simply,

$$U = U_M + U_m = -Mg(r - l) \quad (6)$$

3) *Find Lagrangian of the system:* The Lagrangian of the system is given as  $L \equiv T - U$ . Using Equations 5 and 6, we get

$$\begin{aligned} L &= \left( \frac{1}{2} \dot{r}^2 (m + M) + \frac{1}{2} m r^2 \dot{\theta}^2 \right) - (-Mg(r - l)) \\ L &= \frac{1}{2} \dot{r}^2 (m + M) + \frac{1}{2} m r^2 \dot{\theta}^2 + Mg(r - l) \end{aligned}$$

4) *Find equations of motion for the system:* Using Equations 1 and 2 to find the equations of motion.

For the equations of motion with respect to  $r$ , we have,

$$\frac{\partial L}{\partial r} = m r \dot{\theta}^2 - Mgl \quad (7)$$

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{r}} \right) = \frac{d}{dt} (\dot{r}(m + M)) = \ddot{r}(m + M) \quad (8)$$

And for the equations of motion with respect to  $\theta$ , we have,

$$\frac{\partial L}{\partial \theta} = 0 \quad (9)$$

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{\theta}} \right) = \frac{d}{dt} (m r^2 \dot{\theta}) \quad (10)$$

While it is possible to show the expanded differentiated form of Equation 10, we have left it as is in order to aid our Discussion better.

### B. Approach for Problem 2

1) *Restate objective in technical terms:* Like Problem 1, we first find  $T$  and  $U$  in order to give us  $L \equiv T - U$ . After that we solve for the following equations of motion,

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{x}_1} \right) = \frac{\partial L}{\partial x_1} \quad (11)$$

and

$$\frac{d}{dt} \left( \frac{\partial L}{\partial \dot{x}_2} \right) = \frac{\partial L}{\partial x_2} \quad (12)$$

Where  $x_1$  and  $x_2$  are the displacements as shown in Figure 2.

### C. Find overall kinematic energy of system

Let us first establish our vectors. As shown in the diagram,

## III. RESULTS

### A. Result for Problem 3–2

For Problem 3–2, we obtained our final result by multiplying all three  $H$  matrices that we obtained in the Materials and Methods section. Hence, we obtained the following matrix,

$$H_3^0 = \begin{bmatrix} \sigma_1 & 0 & \sigma_5 & \lambda_1 \\ 0 & 1 & 0 & 0 \\ \sigma_4 & 0 & \sigma_1 & \lambda_2 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

where,

$$\begin{aligned} \lambda_1 &= L_1 \cos(\theta_1) + L_3 \cos(\theta_3) \sigma_2 \\ &\quad - L_3 \sin(\theta_3) \sigma_3 + L_2 \cos(\theta_1) \cos(\theta_2) \\ &\quad - L_2 \sin(\theta_1) \sin(\theta_2) \end{aligned}$$

$$\begin{aligned} \lambda_2 &= -L_1 \sin(\theta_1) - L_3 \cos(\theta_3) \sigma_3 - L_3 \sin(\theta_3) \sigma_2 \\ &\quad - L_2 \cos(\theta_1) \sin(\theta_2) - L_2 \cos(\theta_2) \sin(\theta_1) \end{aligned}$$

and,

$$\begin{aligned} \sigma_1 &= \cos(\theta_3) \sigma_2 - \sin(\theta_3) \sigma_3 \\ \sigma_2 &= \cos(\theta_1) \cos(\theta_2) - \sin(\theta_1) \sin(\theta_2) \\ \sigma_3 &= \cos(\theta_1) \sin(\theta_2) + \cos(\theta_2) \sin(\theta_1) \\ \sigma_4 &= -\cos(\theta_3) \sigma_3 - \sin(\theta_3) \sigma_2 \\ \sigma_5 &= \cos(\theta_3) \sigma_3 + \sin(\theta_3) \sigma_2 \end{aligned}$$

The significance of this  $H_3^0$  matrix is that it provides a direct transformation matrix between the base frame (frame 0) and end-effector frame, which is a solution for forward kinematics.

### B. Result for Problem 3–5

For Problem 3–5, we obtained our final result by multiplying all three  $A_i$  matrices that we obtained in the Materials and Methods section. Hence, we obtained the following matrix,

$$T_3^0 = \begin{bmatrix} \sigma_1 & \sigma_4 & 0 & \lambda_1 \\ \sigma_5 & \sigma_1 & 0 & \lambda_2 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix}$$

where,

$$\lambda_1 = L_3 \cos(\theta_1) \sigma_3 - \sin(\theta_1) (L_1 + q_2) - L_3 \sin(\theta_1) \sigma_2$$

$$\lambda_2 = \cos(\theta_1) (L_1 + q_2) + L_3 \cos(\theta_1) \sigma_2 + L_3 \sigma_3 \sin(\theta_1)$$

and,

$$\sigma_1 = \cos(\theta_1) \sigma_3 - \sin(\theta_1) \sigma_2$$

$$\sigma_2 = \sin\left(\theta_3 + \frac{\pi}{2}\right)$$

$$\sigma_3 = \cos\left(\theta_3 + \frac{\pi}{2}\right)$$

$$\sigma_4 = -\cos(\theta_1) \sigma_2 - \sigma_3 \sin(\theta_1)$$

$$\sigma_5 = \cos(\theta_1) \sigma_2 + \sigma_3 \sin(\theta_1)$$

The significance of this  $T_3^0$  matrix is that it provides a direct transformation matrix between the base frame (frame 0) and end-effector frame, which is a solution for forward kinematics.

### IV. DISCUSSION

In the opinion of the author, this homework problem set was insightful. The first problem proved that we do not need to always use the DH convention when solving for forward kinematics in robotic manipulators. In fact, when using tools like MATLAB, manually executing a non-DH method of computing the forward kinematics is no more complex than using the DH method itself.

The second problem reinforced our learnings from RBE 500. We used the DH convention heavily in that class, so it was great to revisit that foundation as we move forward in this class.

A topic for further consideration could be, when would one prefer to use a non-DH method over the DH method? The DH convention can provide a minimal and efficient way to represent and compute the relationship between the base frame and the end effector in many cases, because it reduces the number of variables involved from 6 to 4. However, suppose we want to model the differential kinematics of a manipulator. The screw-based theory [?] can provide advantages in such a case. In the referenced paper for screw-based theory, it was found that, when various kinematic modelings for common manipulator configurations were compared, the screw-based theory did not provide any disadvantages in any case. The one noticeable difference was that it provided superior flexibility when differential kinematics was compared. The parameter identification is also a bit simpler in the screw-based theory, as compared to the DH-convention.

It was a great exercise to solve this week's problem set, and the author thanks the Professor for this.