

Improve an Engine Cooling Fan: Design for Six Sigma

This example shows how to improve the performance of an engine cooling fan through a **Design for Six Sigma approach using Define, Measure, Analyze, Improve, and Control (DMAIC)**. The initial fan does not circulate enough air through the radiator to keep the engine cool during difficult conditions. First, the example shows how to design an experiment to investigate the effect of three performance factors: *fan distance from the radiator*, *blade-tip clearance*, and *blade pitch angle*. It then shows how to estimate optimum values for each factor, resulting in a design that produces airflows beyond the goal of 875 ft³/min using test data. Finally, it shows how to use simulations to verify that the new design produces airflow according to the specifications in more than 99.999% of the fans manufactured. This example uses MATLAB®, Statistics and Machine Learning Toolbox™, and Optimization Toolbox™.

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1. Define the Problem

This example addresses an engine cooling fan design that is unable to pull enough air through the radiator to keep the engine cool during difficult conditions, such as stop-and-go traffic or hot weather). Suppose you estimate that you need airflow of at least 875 ft³/min to keep the engine cool during difficult conditions. You need to evaluate the current design and develop an alternative design that can achieve the target airflow.

2. Assess, Analyse Cooling Fan Performance: No/Low-Code Approaches

Import originalfan.xlsx as numeric data using importData APP on the HOME tab ribbon or:

```
uiimport % can also "drag 'n drop" input data file into the workspace
```

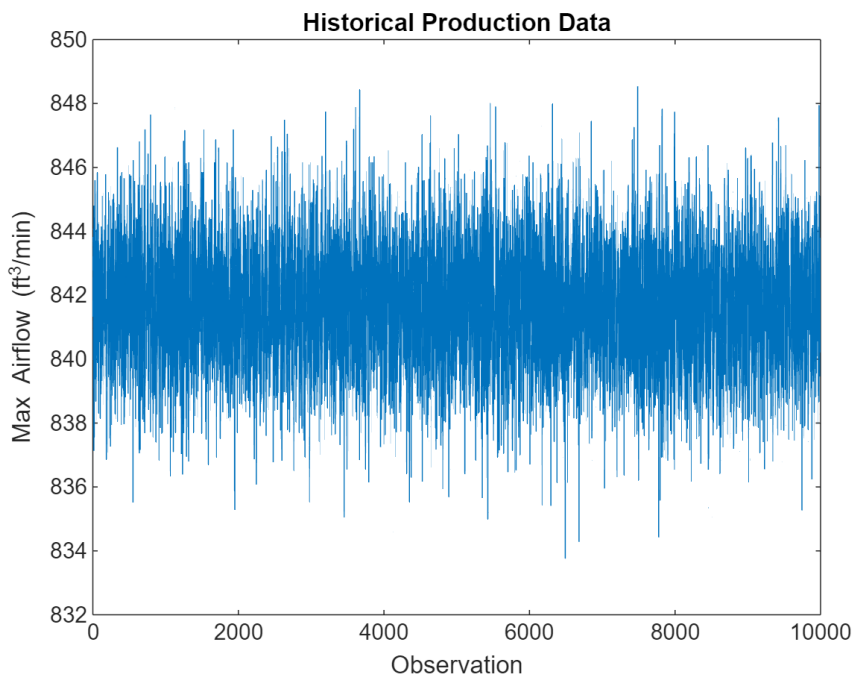
The data consists of 10,000 measurements (historical production data) of the existing cooling fan performance.

Plot the data to analyze the current fan's performance.

```
% Create plot of originalfan
h2 = plot(originalfan,"DisplayName","originalfan");

% Add ylabel, title, and legend
ylabel("originalfan")
title("originalfan")
legend
```

```
plot(originalfan)
xlabel('Observation')
ylabel('Max Airflow (ft^3/min)')
title('Historical Production Data')
```



The data is centered around 842 ft³/min and most values fall within the range of about 8 ft³/min. The plot does not tell much about the underlying distribution of data, however.

NOTE: If data needs to be conditioned, then map the data to a table and use the dataCleaner APP.

```
T = array2table(originalfan)
```

T = 10000x1 table

| | originalfan |
|---|-------------|
| 1 | 844.0720 |
| 2 | 842.1433 |
| 3 | 841.3500 |
| 4 | 841.7039 |

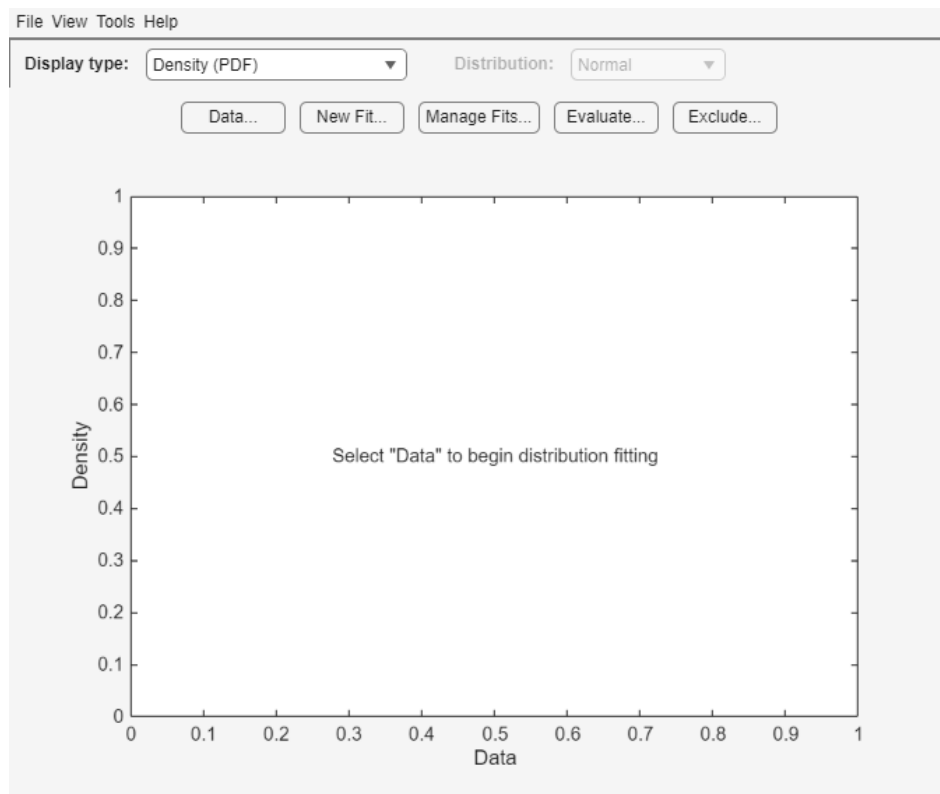
| | originalfan |
|----|-------------|
| 5 | 843.4937 |
| 6 | 839.0935 |
| 7 | 843.4710 |
| 8 | 839.9826 |
| 9 | 840.9740 |
| 10 | 841.3729 |
| 11 | 839.1781 |
| 12 | 840.4986 |
| 13 | 842.4755 |
| 14 | 837.1298 |

⋮

dataCleaner % can also open app directly from the 'Home' tab ribbon

Use the distributionFitter APP to assess the data distribution.

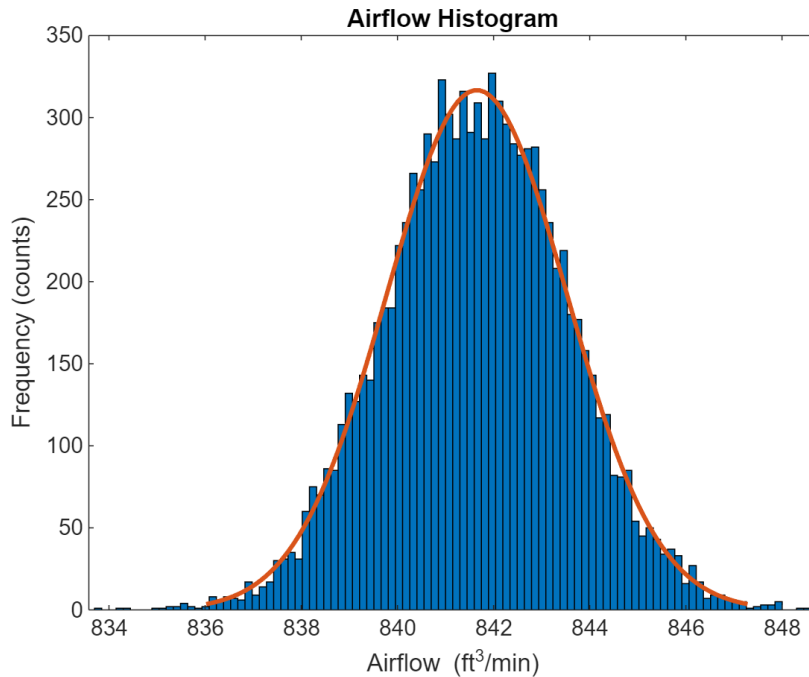
distributionFitter



Alternatively:

Plot the histogram and fit a normal distribution to the data.

```
histfit(originalfan) % Plot histogram with normal distribution fit
xlabel('Airflow (ft3/min)')
ylabel('Frequency (counts)')
title('Airflow Histogram')
```



```
pd = fitdist(originalfan,'normal') % Fit normal distribution to data
```

```
pd =  
NormalDistribution  
  
Normal distribution  
mu = 841.652 [841.616, 841.689]  
sigma = 1.8768 [1.85114, 1.90318]
```

`fitdist` fits a normal distribution to data and estimates the parameters from data. The estimate for the mean airflow speed is 841.652 ft³/min, and the 95% confidence interval for the mean airflow speed is (841.616, 841.689). This estimate makes it clear that the current fan is not close to the required 875 ft³/min. There is need to improve the fan design to achieve the target airflow.

3. Design-Of-Experiments: Determine Factors That Affect Performance

Evaluate the factors that affect cooling fan performance using **design of experiments (DOE)**. The response is the cooling fan airflow rate (ft³/min). Suppose that the factors that you can modify and control are:

- Distance from radiator
- Pitch angle

- **Blade tip clearance**

In general, fluid systems have nonlinear behavior. Therefore, use a response surface design to estimate any nonlinear interactions among the factors. Generate the experimental runs for a [Box-Behnken design](#) in coded (normalized) variables [-1, 0, +1].

```
CodedValue = bbdesign(3)
```

```
CodedValue = 15x3
   -1   -1    0
   -1    1    0
    1   -1    0
    1    1    0
   -1    0   -1
   -1    0    1
    1    0   -1
    1    0    1
    0   -1   -1
    0   -1    1
    ⋮
    ⋮
```

The first column is for the distance from radiator, the second column is for the pitch angle, and the third column is for the blade tip clearance. Suppose you want to test the effects of the variables at the following minimum and maximum values.

- **Distance from radiator: 1 to 1.5 inches**
- **Pitch angle: 15 to 35 degrees**
- **Blade tip clearance: 1 to 2 inches**

Randomize the order of the runs, convert the coded design values to real-world units, and perform the experiment in the order specified.

```
runorder = randperm(15); % Random permutation of the runs
bounds = [1 1.5;15 35;1 2]; % Min and max values for each factor
format shortg

RealValue = zeros(size(CodedValue));
for i = 1:size(CodedValue,2) % Convert coded values to real-world units
    zmax = max(CodedValue(:,i));
    zmin = min(CodedValue(:,i));
    RealValue(:,i) = interp1([zmin zmax],bounds(i,:),CodedValue(:,i));
end
disp(RealValue)
```

```
    1    15    1.5
    1    35    1.5
   1.5    15    1.5
   1.5    35    1.5
    1    25     1
    1    25     2
   1.5    25     1
   1.5    25     2
```

| | | |
|------|----|-----|
| 1.25 | 15 | 1 |
| 1.25 | 15 | 2 |
| 1.25 | 35 | 1 |
| 1.25 | 35 | 2 |
| 1.25 | 25 | 1.5 |
| 1.25 | 25 | 1.5 |
| 1.25 | 25 | 1.5 |

Suppose the at the end of the experiments, you collect the following response values in the variable `TestResult`.

```
TestResult = [837 864 829 856 880 879 872 874 834 833 860 859 874 876 875]';
```

Display the design values and the response.

```
disp(array2table(sortrows([runorder' RealValue TestResult]),...
    'VariableNames',{'RunNumber','Distance','Pitch','Clearance','Airflow'}))
```

| RunNumber | Distance | Pitch | Clearance | Airflow |
|-----------|----------|-------|-----------|---------|
| 1 | 1.25 | 15 | 1 | 834 |
| 2 | 1.25 | 15 | 2 | 833 |
| 3 | 1 | 35 | 1.5 | 864 |
| 4 | 1.25 | 35 | 1 | 860 |
| 5 | 1.5 | 25 | 1 | 872 |
| 6 | 1 | 15 | 1.5 | 837 |
| 7 | 1.5 | 35 | 1.5 | 856 |
| 8 | 1 | 25 | 2 | 879 |
| 9 | 1.25 | 25 | 1.5 | 874 |
| 10 | 1.25 | 25 | 1.5 | 876 |
| 11 | 1.5 | 15 | 1.5 | 829 |
| 12 | 1.25 | 25 | 1.5 | 875 |
| 13 | 1.25 | 35 | 2 | 859 |
| 14 | 1 | 25 | 1 | 880 |
| 15 | 1.5 | 25 | 2 | 874 |

Save the design values and the response in a table.

```
Expmt = table(runorder', CodedValue(:,1), CodedValue(:,2), CodedValue(:,3), ...
    TestResult, 'VariableNames',{'RunNumber','D','P','C','Airflow'})
```

Expmt = 15x5 table

| | RunNumber | D | P | C | Airflow |
|---|-----------|----|----|----|---------|
| 1 | 6 | -1 | -1 | 0 | 837 |
| 2 | 3 | -1 | 1 | 0 | 864 |
| 3 | 11 | 1 | -1 | 0 | 829 |
| 4 | 7 | 1 | 1 | 0 | 856 |
| 5 | 14 | -1 | 0 | -1 | 880 |
| 6 | 8 | -1 | 0 | 1 | 879 |
| 7 | 5 | 1 | 0 | -1 | 872 |

| | RunNumber | D | P | C | Airflow |
|----|-----------|---|----|----|---------|
| 8 | 15 | 1 | 0 | 1 | 874 |
| 9 | 1 | 0 | -1 | -1 | 834 |
| 10 | 2 | 0 | -1 | 1 | 833 |
| 11 | 4 | 0 | 1 | -1 | 860 |
| 12 | 13 | 0 | 1 | 1 | 859 |
| 13 | 9 | 0 | 0 | 0 | 874 |
| 14 | 10 | 0 | 0 | 0 | 876 |

⋮

D stands for Distance, P stands for Pitch, and C stands for Clearance. Based on the experimental test results, the airflow rate is sensitive to the changing factors values. Also, four experimental runs meet or exceed the target airflow rate of 875 ft³/min (runs 2, 4, 12, and 14). However, it is not clear which, if any, of these runs is the optimal one. In addition, it is not obvious how robust the design is to variation in the factors. Create a model based on the current experimental data and use the model to estimate the optimal factor settings.

4. Optimised Modelling: Improve the Cooling Fan Performance

Fit Model

The Box-Behnken design enables you to test for nonlinear (quadratic) effects. The form of the quadratic model is:

$$AF = \beta_0 + \beta_1 * Distance + \beta_2 * Pitch + \beta_3 * Clearance + \beta_4 * Distance * Pitch + \beta_5 * Distance * Clearance + \beta_6 * Pitch * Clearance + \beta_7 * Distance^2 + \beta_8 * Pitch^2 + \beta_9 * Clearance^2$$

where AF is the airflow rate and B_i is the coefficient for the term i . Estimate the coefficients of this model using the `fitlm` function from Statistics and Machine Learning Toolbox.

```
mdl = fitlm(Expmt(:,2:5), 'Airflow~D*P*C-D:P:C+D^2+P^2+C^2')
```

```
mdl =  
Linear regression model:  
Airflow ~ 1 + D*P + D*C + P*C + D^2 + P^2 + C^2
```

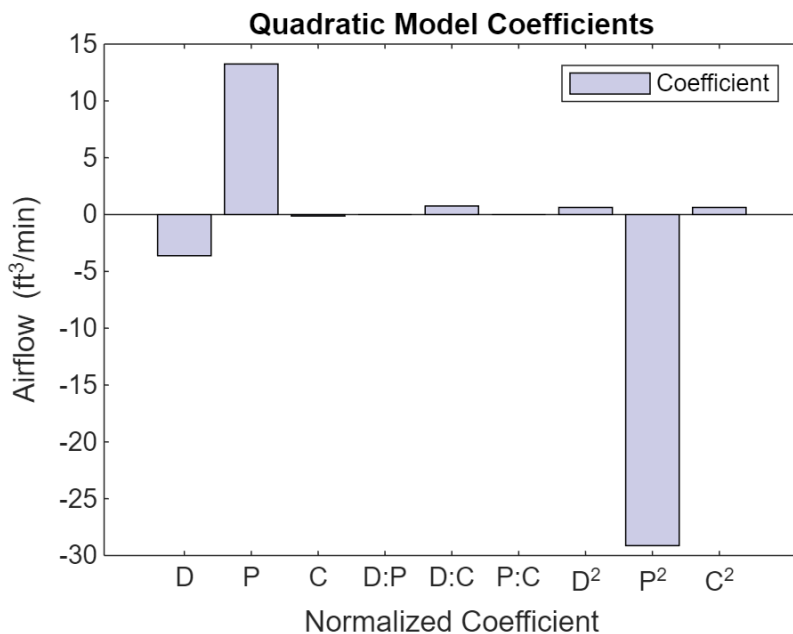
Estimated Coefficients:

| | Estimate | SE | tStat | pValue |
|-------------|------------|---------|------------|------------|
| (Intercept) | 875 | 0.56273 | 1554.9 | 2.0882e-15 |
| D | -3.625 | 0.3446 | -10.519 | 0.00013403 |
| P | 13.25 | 0.3446 | 38.45 | 2.2422e-07 |
| C | -0.125 | 0.3446 | -0.36274 | 0.73163 |
| D:P | 2.2723e-13 | 0.48734 | 4.6628e-13 | 1 |
| D:C | 0.75 | 0.48734 | 1.539 | 0.18443 |
| P:C | 1.6168e-15 | 0.48734 | 3.3175e-15 | 1 |
| D^2 | 0.625 | 0.50724 | 1.2322 | 0.27267 |
| P^2 | -29.125 | 0.50724 | -57.419 | 3.0313e-08 |
| C^2 | 0.625 | 0.50724 | 1.2322 | 0.27267 |

Number of observations: 15, Error degrees of freedom: 5
 Root Mean Squared Error: 0.975
 R-squared: 0.999, Adjusted R-Squared: 0.997
 F-statistic vs. constant model: 550, p-value = 6.01e-07

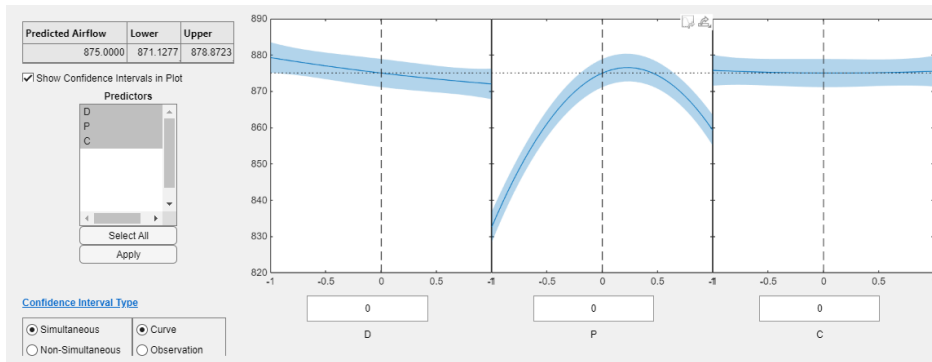
Display the magnitudes of the coefficients (for normalized values) in a bar chart.

```
figure('Units','normalized','Position',[0.05 0.4 0.35 0.4])
h = bar mdl.Coefficients.Estimate(2:10));
set(h,'FaceColor',[0.8 0.8 0.9]);
legend('Coefficient')
set(gca,'XTickLabel',mdl.CoefficientNames(2:10));
ylabel('Airflow (ft^3/min)')
xlabel('Normalized Coefficient')
title('Quadratic Model Coefficients')
```



The bar chart shows that *Pitch* and *Pitch*² are dominant factors. You can look at the relationship between multiple input variables and one output variable by generating a response surface plot. Use [plotSlice](#) to generate response surface plots for the model `mdl` interactively.

```
plotSlice(mdl)
```

The plot shows the nonlinear relationship of airflow with pitch. Move the blue dashed lines around and see the effect the different factors have on airflow.

Optimise Model

Although you can use `plotSlice` to determine the optimum factor settings, you can also use Optimization Toolbox to automate the task.

Find the optimal factor settings using the constrained optimization function `fmincon`.

Write the objective function using `predict mdl, x`;

The objective function is a quadratic response surface fit to the data. We are looking to maximize the objective function. The constraints are the upper and lower limits tested (in coded values) `[-1, 1]`. Set the initial starting point to be the center of the design of the experimental test matrix.

```
prob = optimproblem("ObjectiveSense","maximize");
x = optimvar('optimFactors',1,3,'LowerBound',-1,'UpperBound',1);
x0.optimFactors = zeros(1,3); % Starting point
prob.Objective = fcn2optimexpr(@predict,mdl,x); % Converts function to optimization
expression
options = optimoptions('fmincon','display','iter');
[sol, fval] = solve(prob,x0,'Options',options); % Invoke the solver
```

Solving problem using `fmincon`.

| Iter | F-count | f(x) | Feasibility | First-order optimality | Norm of step |
|------|---------|--------------|-------------|------------------------|--------------|
| 0 | 4 | 8.750000e+02 | 0.000e+00 | 8.833e+00 | |
| 1 | 9 | 8.756049e+02 | 0.000e+00 | 2.015e+01 | 5.905e-01 |
| 2 | 13 | 8.801030e+02 | 0.000e+00 | 9.478e+00 | 6.986e-01 |
| 3 | 17 | 8.819543e+02 | 0.000e+00 | 4.347e+00 | 8.551e-01 |
| 4 | 22 | 8.816620e+02 | 0.000e+00 | 4.628e+00 | 1.438e-01 |
| 5 | 27 | 8.818495e+02 | 0.000e+00 | 4.162e+00 | 7.405e-02 |
| 6 | 31 | 8.819863e+02 | 0.000e+00 | 3.613e+00 | 4.544e-02 |
| 7 | 35 | 8.820231e+02 | 0.000e+00 | 2.580e+00 | 2.208e-02 |
| 8 | 39 | 8.820596e+02 | 0.000e+00 | 9.964e-02 | 4.436e-02 |
| 9 | 43 | 8.822544e+02 | 0.000e+00 | 7.096e-02 | 4.930e-02 |
| 10 | 47 | 8.822566e+02 | 0.000e+00 | 6.718e-03 | 7.182e-04 |
| 11 | 51 | 8.822566e+02 | 0.000e+00 | 5.367e-03 | 3.279e-05 |
| 12 | 55 | 8.822566e+02 | 0.000e+00 | 4.622e-04 | 8.413e-05 |
| 13 | 59 | 8.822569e+02 | 0.000e+00 | 1.202e-04 | 8.077e-05 |
| 14 | 63 | 8.822570e+02 | 0.000e+00 | 9.136e-05 | 1.996e-05 |
| 15 | 67 | 8.822570e+02 | 0.000e+00 | 8.373e-05 | 7.106e-08 |
| 16 | 71 | 8.822570e+02 | 0.000e+00 | 6.847e-05 | 3.227e-07 |

| | | | | | |
|----|----|--------------|-----------|-----------|-----------|
| 17 | 75 | 8.822570e+02 | 0.000e+00 | 2.757e-05 | 1.319e-06 |
| 18 | 79 | 8.822570e+02 | 0.000e+00 | 4.000e-07 | 1.352e-07 |

Feasible point with lower objective function value found, but optimality criteria not satisfied. See output.bestfeas

Local minimum found that satisfies the constraints.

Optimization completed because the objective function is non-decreasing in feasible directions, to within the value of the optimality tolerance, and constraints are satisfied to within the value of the constraint tolerance.

<stopping criteria details>

Convert the results to a maximization problem and real-world units.

```
maxval = fval;
optfactors = sol.optimFactors;
maxloc = (optfactors + 1)';
bounds = [1 1.5;15 35;1 2];
maxloc=bounds(:,1)+maxloc .* ((bounds(:,2) - bounds(:,1))/2);
disp(array2table([maxloc' maxval], 'VariableNames',...
    {'Distance', 'Pitch', 'Clearance', 'Airflow'}))
```

| Distance | Pitch | Clearance | Airflow |
|----------|--------|-----------|---------|
| 1 | 27.275 | 1 | 882.26 |

The optimization result suggests placing the new fan one inch from the radiator, with a one-inch clearance between the tips of the fan blades and the shroud.

Because pitch angle has such a significant effect on airflow, perform additional analysis to verify that a 27.3 degree pitch angle is optimal.

```
load 'Data/AirflowData.mat'
tbl = table(pitch,airflow);
mdl2 = fitlm(tbl, 'airflow~pitch^2')
```

```
mdl2 =
Linear regression model:
    airflow ~ 1 + pitch + pitch^2
```

Estimated Coefficients:

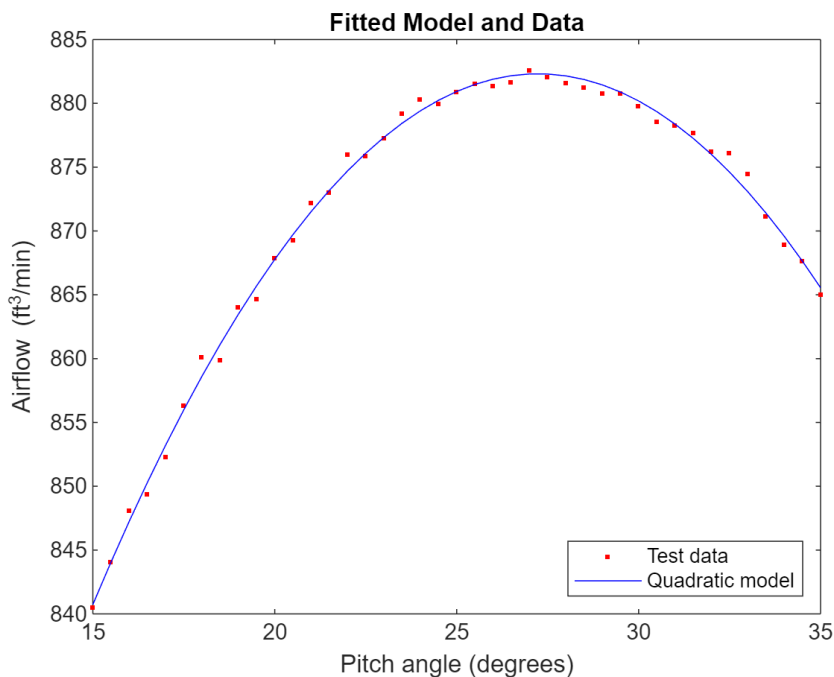
| | Estimate | SE | tStat | pValue |
|-------------|----------|-----------|---------|------------|
| (Intercept) | 676.23 | 2.1727 | 311.24 | 2.4225e-66 |
| pitch | 15.135 | 0.18043 | 83.882 | 9.6031e-45 |
| pitch^2 | -0.27788 | 0.0035887 | -77.433 | 1.9732e-43 |

```
Number of observations: 41, Error degrees of freedom: 38
Root Mean Squared Error: 0.719
R-squared: 0.996, Adjusted R-Squared: 0.996
F-statistic vs. constant model: 5.14e+03, p-value = 5.78e-47
```

The results show that a quadratic model explains the effect of pitch on the airflow well.

Plot the pitch angle against airflow and impose the fitted model.

```
figure
plot(pitch,airflow,'.r')
hold on
ylim([840 885])
line(pitch,mdl2.Fitted,'color','b')
title('Fitted Model and Data')
xlabel('Pitch angle (degrees)')
ylabel('Airflow (ft^3/min)')
legend('Test data','Quadratic model','Location','se')
hold off
```



Find the pitch value that corresponds to the maximum airflow.

```
[~, idx] = max(airflow);
disp(pitch(idx))
```

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The additional analysis confirms that a 27.3 degree pitch angle is optimal.

The improved cooling fan design meets the airflow requirements. You also have a model that approximates the fan performance well based on the factors you can modify in the design. Ensure that the fan performance is robust to variability in manufacturing and installation by performing a sensitivity analysis.

5. Assess Model: Sensitivity Analysis

Suppose that, based on historical experience, the manufacturing uncertainty is as follows.

Factor: Real Values, Coded Values

- Distance from radiator: 1.00 +/- 0.05 inch, 1.00 +/- 0.20 inch
- Blade pitch angle: 27.3 +/- 0.25 degrees, 0.227 +/- 0.028 degrees
- Blade tip clearance: 1.00 +/- 0.125 inch, -1.00 +/- 0.25 inch

Verify that these variations in factors will enable to maintain a robust design around the target airflow. The philosophy of Six Sigma targets a defect rate of no more than 3.4 per 1,000,000 fans. That is, the fans must hit the 875 ft³/min target 99.999% of the time.

You can verify the design using Monte Carlo simulation. Generate 10,000 random numbers for three factors with the specified tolerance. First, set the state of the random number generators so results are consistent across different runs.

```
rng('default')
```

Perform the Monte Carlo simulation. Include a noise variable that is proportional to the noise in the fitted model, mdl (that is, the RMS error of the model). Because the model coefficients are in coded variables, you must generate dist, pitch, and clearance using the coded definition.

```
dist = random('normal',optfactors(1),0.20,[10000 1]);  
pitch = random('normal',optfactors(2),0.028,[10000 1]);  
clearance = random('normal',optfactors(3),0.25,[10000 1]);  
noise = random('normal',0,mdl2.RMSE,[10000 1]);
```

Calculate airflow for 10,000 random factor combinations using the model.

```
simfactor = [dist pitch clearance];  
X = predict(mdl,simfactor);
```

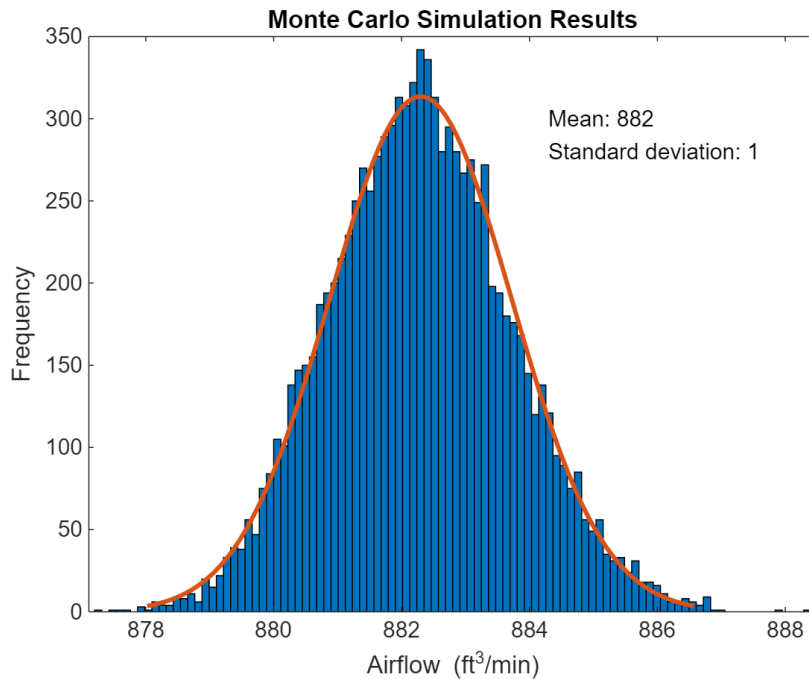
Add noise to the model (the variation in the data that the model did not account for).

```
simflow = X + noise;
```

Evaluate the variation in the model's predicted airflow using a histogram. To estimate the mean and standard deviation, fit a normal distribution to data.

```
pd = fitdist(simflow,'normal');  
histfit(simflow)  
hold on  
text(pd.mu+2,300,['Mean: ' num2str(round(pd.mu))])  
text(pd.mu+2,280,['Standard deviation: ' num2str(round(pd.sigma))])  
hold off
```

```
xlabel('Airflow (ft^3/min)')
ylabel('Frequency')
title('Monte Carlo Simulation Results')
```



The results look promising. The average airflow is 882 ft³/min and appears to be better than 875 ft³/min for most of the data.

Determine the probability that the airflow is at 875 ft³/min or below.

```
format long
pfail = cdf(pd,875)
```

```
pfail =
1.509288967210659e-07
```

```
pass = (1-pfail)*100
```

```
pass =
99.999984907110331
```

The design appears to achieve at least 875 ft³/min of airflow 99.999% of the time.

Use the simulation results to estimate the process capability.

```
S = capability(simflow,[875.0 890])
```

```
S = struct with fields:
    mu: 8.822982645663780e+02
    sigma: 1.424806875428719
    P: 0.999999816749821
    Pl: 1.509288967210659e-07
```

```
Pu: 3.232128233668679e-08
Cp: 1.754623762078464
Cpl: 1.707427790680271
Cpu: 1.801819733476657
Cpk: 1.707427790680271
```

```
pass = (1-S.Pl)*100
```

```
pass =
    99.999984907110331
```

The Cp value is 1.75. A process is considered high quality when Cp is greater than or equal to 1.6. The Cpk is similar to the Cp value, which indicates that the process is centered. Now implement this design. Monitor it to verify the design process and to ensure that the cooling fan delivers high-quality performance.

6. Evaluate Model: Control Manufacturing of the Improved Cooling Fan

You can monitor and evaluate the manufacturing and installation process of the new fan using control charts. Evaluate the first 30 days of production of the new cooling fan. Initially, five cooling fans per day were produced. First, load the sample data from the new process.

```
load 'Data/spcdata.mat'
```

Plot the X-bar and S charts.

```
figure
controlchart(spcflow, 'chart', {'xbar', 's'}) % Reshape the data into daily sets
xlabel('Day')
```

According to the results, the manufacturing process is in statistical control, as indicated by the absence of violations of control limits or nonrandom patterns in the data over time.

You can also run a capability analysis on the data to evaluate the process.

```
S2 = capability(spcflow(:), [875.0 890])
pass = (1-S.Pl)*100
```

The Cp value of 1.755 is very similar to the estimated value of 1.73. The Cpk value of 1.66 is smaller than the Cp value. However, only a Cpk value less than 1.33, which indicates that the process shifted significantly toward one of the process limits, is a concern. The process is well within the limits and it achieves the target airflow (875 ft³/min) more than 99.999% of the time.

Multi-Criteria Decision Making

7. Technique for Order Preference by Similarity to Ideal Solution (TOPSIS)

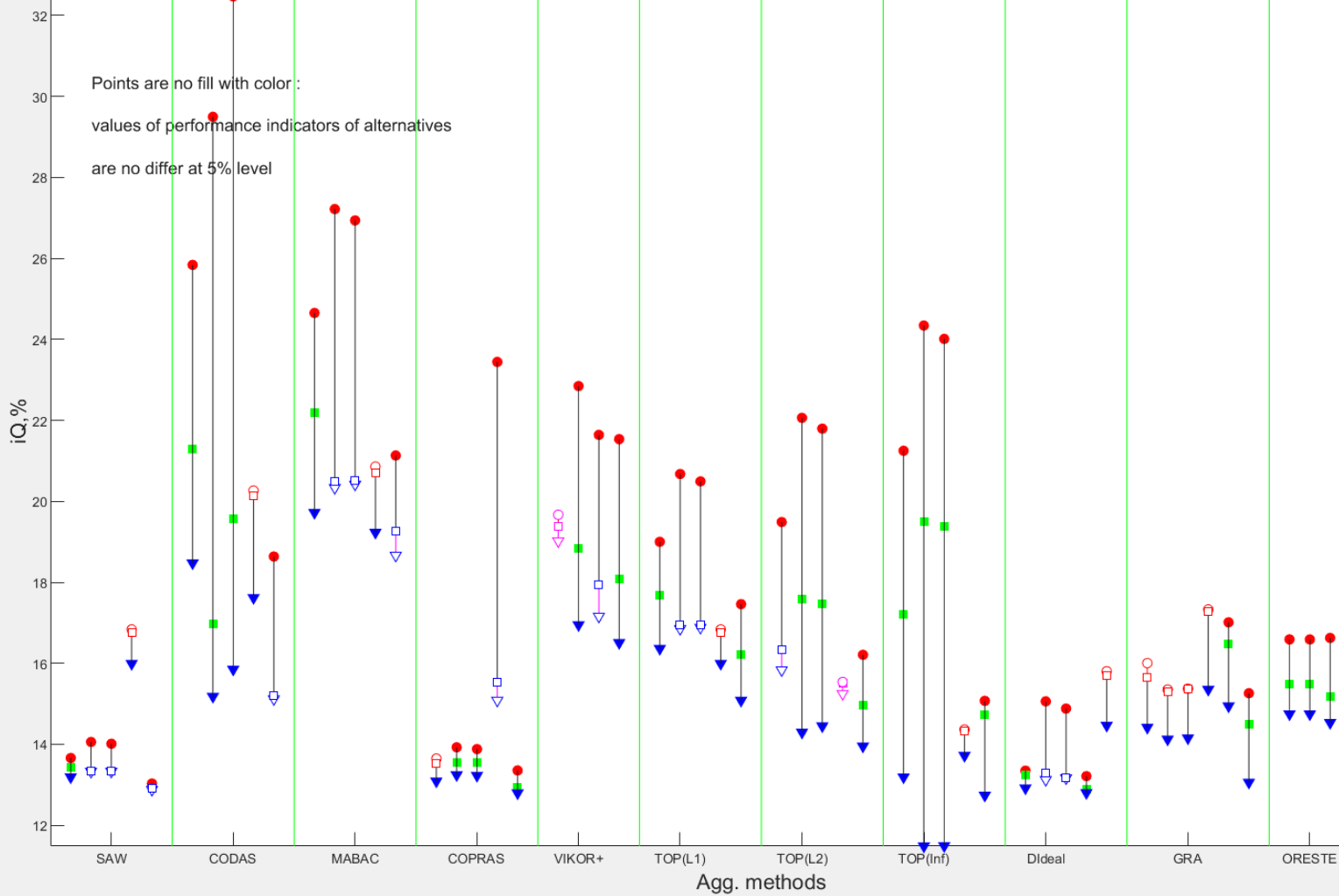
The TOPSIS (Technique for Order of Preference by Similarity to Ideal Solution) method is a multi-criteria decision-making technique used to evaluate and rank alternatives based on their similarity to ideal solutions. It works by normalizing scores for each criterion and calculating the geometric distance between each alternative and the ideal solution, with the goal of minimizing distance to the ideal and maximizing distance from the negative ideal. TOPSIS is particularly useful in scenarios where the decision-maker needs to balance multiple criteria and where the data is not fully defined or precise. The technique provides a structured approach to decision-making, allowing for the selection of the most suitable alternative based on the criteria set. As such, it is widely applied in various fields, including business, engineering, and healthcare, to handle complex decision-making problems by allowing trade-offs between conflicting objectives. In particular, this method is applied in ranking machine learning models on basis of various factors like correlation, R^2 , RMS-error, accuracy, precision, F-score, etc.

Algorithm

1. **Define the Problem:** Identify problem and determine the evaluation criteria for decision alternatives.
2. **Construct a Decision Matrix:** Create a matrix with alternatives as rows and criteria as columns.
3. **Normalize the Decision Matrix:** Ensure each criterion receives equal weight by normalizing the values.
4. **Determine the Weighted Normalized Decision Matrix:** Assign weights to each criterion based on their importance.
5. **Determine the Positive and Negative Ideal Solutions:** Identify the best and worst possible values for each criterion.
6. **Calculate the Separation Measures:** Compute the Euclidean distance of each alternative from the positive and negative ideal solutions.
7. **Calculate the Relative Closeness to the Ideal Solution:** Determine the closeness of each alternative to the ideal solution and rank the alternatives based on these values.

| TOPSIS - L1 Norm | | | |
|------------------|--|---|--------------|
| RANK | Q - Performance Indicator for Alternatives | iQ - Intensity of Performance Indicator of Alternatives | R - Rank for |
| 1 | 0.707 | 22.85 | |
| 2 | 0.583 | 18.85 | |
| 3 | 0.524 | 16.95 | |
| 4 | 0.392 | 12.68 | |
| 5 | 0.306 | 9.88 | |
| 6 | 0.268 | 8.66 | |
| 7 | 0.168 | 5.42 | |
| 8 | 0.146 | 4.71 | |

**Intensity of Performance indicator (iQ)
for normalization methods: MAX,SUM,VEC,M-M,DEA**



FOR FULL DETAILS: download [MCDM_tools](#) from the File Exchange on MATLAB Central and explore (to generate MCDM examples, run MCDM_tools.m).