# MIE100: Dynamics

## Arnav Patil

Department of Electrical and Computer Engineering, University of Toronto

## Contents

1	Kinematics of Particles	2
2	Kinetics of Particles	3
3	Kinetics of Systems of Particles	4
4	Plane Kinematics of Rigid Bodies	5
5	Plane Kinetics of Rigid Bodies	6
6	Vibration & Time Response	7

## 1 Kinematics of Particles

#### **Rectilinear Motion**

$$v = \frac{ds}{dt} = \dot{s} \tag{1}$$

$$d = \frac{dv}{dt} = \ddot{s} \tag{2}$$

• If we eliminate dt, we get v dv = a ds.

$$\int_{v_1}^{v_2} dv = \int_{t_1}^{t_2} a \cdot dt \tag{3}$$

$$\int_{v_1}^{v_2} v \cdot dv = \int_{s_1}^{s_2} a \cdot ds \ \mathbf{OR} \ \frac{1}{2} (v_2^2 - v_1^2)$$
 (4)

#### Plane Curvilinear Motion

$$v = \lim_{t \to 0} \frac{\Delta r}{\Delta t} \to v = \frac{dr}{dt} = \dot{r} \tag{5}$$

For projectile motion,  $a_x = 0$  and  $a_y = -g$ .

#### Normal and Tangential (n-t) Coordinates

$$v = v\hat{e}_t = \rho \dot{\beta} \hat{e}_t \tag{6}$$

where  $\rho$  is the radius of curvature and  $\beta$  is in radians.

For circular motion,

$$a = \frac{v^2}{r}e_n + \dot{v}e_t \tag{7}$$

$$v = r\dot{\theta}, a_n = \frac{v^2}{r} = r\dot{\theta}^2 = v\dot{\theta}, a_t = \dot{v} = r\ddot{\theta}$$
 (8)

## Polar Coordinates $(r - \theta)$

Velocity:

$$v = \dot{r}\hat{e_r} + r\dot{\theta}\hat{e_\theta} \tag{9}$$

Acceleration:

$$a = (\ddot{r} - r\dot{\theta}^2)\hat{e_r} + (r\ddot{\theta} + 2\dot{r}\dot{\theta})\hat{e_\theta}$$
(10)

For circular motion with a constant r, components become:

- $v_r = 0$
- $v_{\theta} = r\dot{\theta}$
- $\bullet \ a_r = -r\dot{\theta^2}$
- $\bullet \ a_{\theta} = r \ddot{\theta}$

### Relative Motion (Translating Axis)

$$r_B = r_A + r_{B/A} \to v_B = v_A + v_{B/A} \to a_B = a_A + a_{B/A}$$
 (11)

#### 2 Kinetics of Particles

#### Newton's Second Law

$$F = ma (12)$$

#### **Equation of Motion and Solutions of Problems**

$$\sum F = ma \tag{13}$$

Remember to determine if motion is constrained or unconstrained.

#### **Rectilinear Motion**

Split up  $\sum F = ma$  into its directional components.

#### **Curvilinear Motion**

$$\sum F_r = ma_r \to a_r = \ddot{r} - r\dot{\theta}^2 \tag{14}$$

$$\sum F_{\theta} = ma_{\theta} \to a_{\theta} = r\ddot{\theta} + 2\dot{r}\dot{\theta} \tag{15}$$

#### Work and Kinetic Energy

$$U = \int F \cdot dr \tag{16}$$

- 1. Work associated with a constant external force,
- 2. Work associated with a spring force, and
- 3. Work associated with weight.

$$U_{1\to 2} = \int_{1}^{2} F \cdot dr = \int_{v_1}^{v_2} mv \cdot dv = \frac{1}{2} m(v_2^2 - v_1^2)$$
 (17)

$$E_K = \frac{1}{2}mv^2 \tag{18}$$

$$U_{1\to 2} = T_2 - T_1 = \Delta T \tag{19}$$

Power:  $P = F \cdot v$ .

#### Potential Energy

$$V_g = mgh (20)$$

$$\Delta V_e = \frac{1}{2}k(x_2^2 - x_1^2) \tag{21}$$

Work-energy equation:  $T_1 + V_1 + U_{1\rightarrow 2} = T_2 + V_2$ 

#### Linear Impulse and Linear Momentum

$$\sum F = m\dot{v} \to \sum F = \dot{G} \tag{22}$$

Impulse:

$$\int_{t_1}^{t_2} \sum F \cdot dt = G_2 - G_1 = \Delta G \to G_1 + \int_{t_1}^{t_2} \sum F \cdot dt = G_2$$
 (23)

#### Angular Impulse and Angular Momentum

$$H_O = r \times mv \to \sum M_O = \dot{H}_O \tag{24}$$

## 3 Kinetics of Systems of Particles

#### Generalized Newton's Second Law

For a system of particles:  $mr = \sum m_i r_i$  Kinetic energy expression:  $v_i = v + \dot{\rho}_i$ , where  $\rho_i$  is the velocity of  $m_i$  with regards to a translating reference frame moving with the mass centre G.

$$T = \frac{1}{2}mv^2 + \sum_{i=1}^{\infty} \frac{1}{2}m_i|\dot{\rho_i}|$$
 (25)

#### Impulse-Momentum

$$G = mv \to \sum F = \dot{G} \tag{26}$$

Angular momentum:

• About a fixed point O:

$$H_O = \sum (r_i \times m_i v_i) \tag{27}$$

$$\sum m_O = \dot{H_O} \tag{28}$$

• About the mas centre G:

$$H_G = \sum \rho_i \times m_i \dot{r_i} \tag{29}$$

$$\sum M_G = \dot{H_G} \tag{30}$$

• About an arbitrary point P:

$$H_P = H_G + \rho \times mv \tag{31}$$

$$\sum M_P = \dot{H_G} + \rho \times ma \tag{32}$$

## Conservation of Energy and Momentum

$$\Delta T + \Delta V = 0 \to T_1 + V_1 = T_2 + V_2 \tag{33}$$

$$(H_O)_1 = (H_O)_2 \ \mathbf{OR} \ (H_G)_1 = (H_G)_2$$
 (34)

## 4 Plane Kinematics of Rigid Bodies

#### Rotations

$$\omega = \omega_0 + \alpha t \tag{35}$$

$$\omega^2 = \omega_0^2 + 2\alpha(\theta - \theta_0) \tag{36}$$

$$\theta = \theta_0 + \omega_0 t + \frac{1}{2} \alpha t^2 \tag{37}$$

Rotation about a fixed axis:

- $v = \omega \times r$
- $a_n = \omega \times \omega \times r = \frac{v^2}{r} = \omega \times v$
- $a_t = \alpha \times r$

#### Relative Velocity

SINCLAIR'S SPAM EQUATION:  $v_B = v_A + \omega_{AB} \times r_{B/A}$ 

$$\Delta r_A = \Delta r_B + \Delta r_{A/B} \tag{38}$$

## Instantaneous Centre of Zero Velocity

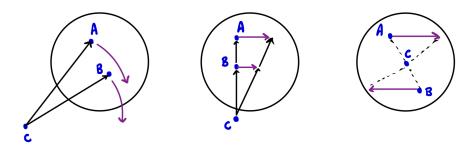


Figure 1: Instantaneous Centre of Zero Velocity

#### Relative Acceleration

$$a_A = a_B + a_{A/B} \tag{39}$$

$$(a_{A/B})_n = \frac{v_{A/B}^2}{r} = r\omega^2 \tag{40}$$

$$(a_{A/B})_t = \dot{v_{A/B}} = r\alpha \tag{41}$$

## 5 Plane Kinetics of Rigid Bodies

#### General Equations of Motion

$$\sum F = ma \to \sum M_G = \dot{H}_G \tag{42}$$

$$H_G = I\omega \rightarrow \sum M_G = \dot{H}_G = I\dot{\omega} = I\alpha$$
 (43)

$$\sum M_P = \dot{H}_G + \rho \times ma \tag{44}$$

This is summed up as:  $M_P = I\alpha + mad$ 

$$\sum M_O = I_O \alpha \tag{45}$$

Analysis procedure:

- 1. Kinematics equations
- 2. Diagrams identify knowns and unknowns
- 3. Equations of motion use to get extra variables and solvable system

### Translation

For a translating body, our general equations of motion are:

$$\sum F = ma \tag{46}$$

$$\sum M_G = I\alpha = 0 \tag{47}$$

#### **Fixed-Axis Rotation**

Almost same equations are applicable here:

$$\sum F = ma \tag{48}$$

$$\sum M_G = I\alpha \tag{49}$$

$$\sum M_O = I_O \alpha \tag{50}$$

#### General Plane Motion

Solving plane motion problems:

- 1. Choice of coordinate system
- 2. Choice of moment equations  $\sum M_P = I\alpha + mad$
- 3. Choice of constrained vs unconstrained motion
- 4. Number of unknowns
- 5. Identify body or system
- 6. Kinematics equations
- 7. Consistency in assumptions

#### Work-Energy Relations

$$U = \int F \cdot dr \ \mathbf{OR} \ U = \int (F \cos \alpha) ds \tag{51}$$

Kinetic energy:

- Translation  $T = \frac{1}{2}mv^2$
- Fixed axis rotation  $T = \frac{1}{2}I_O\omega^2$
- General plane motion  $T = \frac{1}{2}mv^2 + \frac{1}{2}I_O\omega^2$ 
  - Can also be expressed at the IC  $T = \frac{1}{2}I_C\omega^2$

## 6 Vibration & Time Response

#### Free Vibration of Particles

Applying Newton's Second Law in the form  $\sum F_x = m\ddot{x}$ :

- $-kx = m\ddot{x}$  **OR**  $m\ddot{x} + kx = 0$
- Oscillation of a mass objected to a linear restoring force as described by this equation is called SIMPLE HARMONIC MOTION and is characterized by acceleration which is proportional to the displacement but of the opposite sign.
- $\ddot{x} + \omega_n^2 x = 0$ , which gives us  $\omega_n = \sqrt{k/m}$ .

Solution for undamped free motion:

$$x(t) = A\cos\omega_n t + B\sin\omega_n t \tag{52}$$

$$x(t) = C\sin(\omega_n t + \psi) \tag{53}$$

At t = 0, we get  $x_0 = A$ , and  $\dot{x}_0 = B\omega_n$ . If t = 0, then  $x_0 = C\sin\psi$  and  $\dot{x}_0 = C\omega_n\cos\psi$ .

• 
$$\psi = \tan^{-1}(\frac{x_0 \omega_n}{\dot{x}_0})$$

Natural frequency:  $f_n = \frac{\omega_n}{2\pi}$ 

For damped free vibration:

$$m\ddot{x} + c\dot{x} + kx = 0 \tag{54}$$

If we define  $\zeta = \frac{c}{2m\omega_n}$ , then we can say:

$$\ddot{x} + 2\zeta\omega_n\dot{x} + \omega_n^2 x = 0 \tag{55}$$

For free damped vibration:

$$x = Ae^{rt} (56)$$

$$\lambda^2 + 2\zeta\omega_n\lambda + \omega_n^2 = 0 \tag{57}$$

$$x = A_1 e^{r_1 t} + A_2 e^{r_2 t} (58)$$

Categories of damped motion:

- $\zeta > 1$  OVERDAMPED
- $\zeta < 1$  UNDERDAMPED

#### Forced Vibration of Particles

$$\ddot{x} + 2\zeta\omega_n\dot{x} + \omega_n^2 x = \frac{F_0\sin\omega t}{m} \tag{59}$$

#### Vibration of Rigid Bodies

Use same equations derived throughout the course, just add rotation to the calculations.