

HW 8: Mini Computer Project - PWR Modeling

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14 November 2025

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1 Introduction

In this project, we aimed to solve for information regarding a channel of fluid flow in a PWR. Certain variables were provided, and using known conservation equations for the fluid and the general heat diffusion equations within the individual rods, information can be determined for the full system. This information includes temperature anywhere in the rod for a given axial location, the temperature of the fluid based on the axial location, the equilibrium quantity of the fluid, pressure of the fluid, and many more. These calculations were completed using numerical integration methods, using discretization to solve the axial changes without a continuous function. The `PyFluids` package was used to access all of the data for the water properties at different temperatures, pressures, and qualities. Functions were designed to properly access these values, but they allowed for the use of an established interpolation program instead of redundant development.

2 Temperature in Fluid Equations

In a PWR, the liquid should remain sub-cooled the entire time - χ_e should remain below 0 for the entire channel.

The heat generation in the fuel is given with Equation 1 and all of the variables are present in the code.

$$q'(z) = q'_0 \cdot \cos(\pi z / H_e) \left[\frac{W}{m} \right] \quad (1)$$

This can be converted to other measured fluxes for the rod surfaces (q''), or heat generation within the fuel pellet (q''') using the relations in Equation 2.

$$q[W] = q' \left[\frac{W}{m} \right] \cdot \Delta z[m] = q'' \left[\frac{W}{m^2} \right] \cdot (2\pi r_{fuel} \cdot \Delta z)[m^2] = q''' \left[\frac{W}{m^3} \right] \cdot (\pi r_{fuel}^2 \cdot \Delta z)[m^3] \quad (2)$$

This is the mass conservation equation for the fluid, it is used to ensure that G, the momentum of the fluid or ρv , is constant.

$$\frac{d\rho}{dz} + \frac{d\rho v}{dz} = 0$$

This is the momentum conservation equation for the fluid, it is used to

calculate the pressure drop along the channel.

$$\frac{d\rho v}{dt} + \frac{d\rho v v}{dz} = -\frac{dP}{dz} - \frac{\tau_f \xi_w}{A_f} - \rho g \sin(\theta)$$

$$-\frac{dP}{dz} = G^2 \cancel{\frac{d}{dz}} \cancel{\frac{1}{\rho}} + \frac{\tau_f \xi_w}{A_f} + \rho g + \cancel{\frac{d\rho v}{dt}}$$

Where $\tau_f = \frac{1}{2} f \frac{G^2}{\rho}$, and $f = f_{1\phi} = 0.316 Re^{-0.25}$

$$-\frac{dP}{dz} = \frac{1}{2} (0.316 Re^{-0.25}) \frac{G^2 \xi_w}{\rho A_f} + \rho g$$

$$\frac{dP}{dz} = \frac{P^{i+1} - P^i}{\Delta z}$$

This is the energy conservation equation for the fluid, it is used to calculate the change in equilibrium quality along the channel, the corresponding heat transfer coefficients (h), and the change in temperature along the channel.

$$\frac{d}{dt} \rho_m h + \frac{d}{dz} \rho_m v h = \frac{q'' \xi_w}{A_f} + \frac{dP}{dt} + q'''$$

$$\cancel{\frac{d}{dt} \rho_m h} + G \frac{d}{dz} h = \frac{q'' \xi_w}{A_f} + \cancel{\frac{dP}{dt}} + \cancel{q'''}$$

$$\frac{dh}{dz} = \frac{q'' \xi_w}{A_f G}$$

$$h = \chi_e \cdot h_{fg} + h_f$$

$$\frac{dh_f}{dz} + h_{fg} \frac{d\chi_e}{dz} + \chi_e \frac{dh_{fg}}{dz} = \frac{q'' \xi_w}{A_f G}$$

$$\frac{h_f^i - h_f^{i-1}}{\Delta z} + h_{fg}^{i-1} \frac{\chi_e^i - \chi_e^{i-1}}{\Delta z} + \chi_e^{i-1} \frac{h_{fg}^i - h_{fg}^{i-1}}{\Delta z} = \frac{q'' \xi_w}{A_f G}$$

$$\chi_e^i = \frac{1}{h_{fg}^{i-1}} \left(\frac{q'' \xi_w \Delta z}{A_f G} - (h_f^i - h_f^{i-1}) - \chi_e^{i-1} (h_{fg}^i - h_{fg}^{i-1}) \right) + \chi_e^{i-1}$$

3 Temperature in Rod Equations

Region 1 is for $r < r_{fuel}$ corresponding to the fuel pellet where heat is being generated. Region 2 corresponds to the gap between the pellet and clad and Region 3 corresponds to the cladding. The dimensions of these different regions are all given in the code.

$$\begin{aligned}
\nabla k \nabla T_1 &= -q''' \\
\frac{1}{r} \frac{d}{dr} r \frac{dT_1}{dr} &= -\frac{q'(z)}{\pi r_{fuel}^2 \cdot k_{fuel}} \\
\frac{d}{dr} r \frac{dT_1}{dr} &= -\frac{q'(z)}{\pi r_{fuel}^2 \cdot k_{fuel}} r \\
r \frac{dT_1}{dr} &= -\frac{q'(z)}{\pi r_{fuel}^2 \cdot k_{fuel}} \frac{r^2}{2} + C_1 \\
\frac{dT_1}{dr} &= -\frac{q'(z)}{\pi r_{fuel}^2 \cdot k_{fuel}} \frac{r}{2} + \frac{C_1}{r} \\
T_1(r) &= -\frac{q'(z)}{\pi r_{fuel}^2 \cdot k_{fuel}} \frac{r^2}{4} + C_1 \cdot \ln(r) + C_2
\end{aligned}$$

$$\begin{aligned}
\nabla k \nabla T_2 &= 0 \\
\frac{1}{r} \frac{d}{dr} r \frac{dT_2}{dr} &= 0 \\
\frac{dT_2}{dr} &= \frac{C_3}{r} \\
T_2(r) &= C_3 \ln(r) + C_4 \\
\nabla k \nabla T_3 &= 0 \\
\frac{1}{r} \frac{d}{dr} r \frac{dT_3}{dr} &= 0 \\
\frac{dT_3}{dr} &= \frac{C_5}{r} \\
T_3(r) &= C_5 \ln(r) + C_6
\end{aligned}$$

Once general equations are solved for, the boundary conditions can be applied to each vertical slice (Δz) to get the temperatures at each point in the rod.

B.C.s

$$1. \frac{dT_1}{dr}|_{r=0} = 0 \text{ due to symmetry}$$

So, $C_1 = 0$

$$2. -k_{fuel} \frac{dT_1}{dr}|_{r=r_{fuel}} = -k_{gap} \frac{dT_2}{dr}|_{r=r_{fuel}} \\ -k_{fuel} \left(-\frac{q'(z)}{\pi r_{fuel}^2 \cdot k_{fuel}} \frac{r_{fuel}}{2} \right) = -k_{gap} \frac{C_3}{r_{fuel}} \\ \left(\frac{-q'(z)}{2\pi \cdot k_{gap}} \right) = C_3$$

$$3. -k_{gap} \frac{dT_2}{dr}|_{r=r_{c,i}} = -k_{clad} \frac{dT_3}{dr}|_{r=r_{c,i}} \\ -k_{gap} \frac{C_3}{r_{c,i}} = -k_{clad} \frac{C_5}{r_{c,i}} \\ \frac{k_{gap} C_3}{k_{clad}} = C_5$$

This boundary condition is for the interface of the cladding with the fluid. When the wall temperature is less than the saturation temperature, there is a different method of calculating the heat flux and the corresponding necessary constant compared to when the wall temperature is greater than the saturation temperature. Both sections are given below

$$4. (\text{if } T_w < T_{sat}) -k_{clad} \frac{dT_3}{dr}|_{r=r_{c,o}} = h(T_3(r_{c,o}) - T_f) \\ -k_{clad} \frac{C_5}{r_{c,0}} = h(C_5 \ln(r_{c,o}) + C_6 - T_f) \\ C_5 \left(\frac{-k_{clad}}{h \cdot r_{c,0}} - \ln(r_{c,o}) \right) + T_f = C_6$$

$$\begin{aligned}
& 4. (\text{if } T_w \geq T_{sat}) \\
-k_{clad} \frac{dT_3}{dr}|_{r=r_{c,o}} &= q'' = \sqrt{(F \cdot h_{1\phi} \cdot (T_3(r_{c,o}) - T_f))^2 + (S \cdot h_{NB} \cdot (T_3(r_{c,o}) - T_{sat}))^2} \\
F &= (1 + \chi \cdot \Pr \cdot (\frac{\rho_f}{\rho_g} - 1))^{0.35} \\
S &= (1 + 0.055 F^{0.1} Re_L^{0.16})^{-1} \\
h_{NB} &= 55(\frac{P}{P_c})^{0.12}(q'')^{2/3} \cdot (-\log_{10}(\frac{P}{P_c}))^{-0.55} \cdot MM_w^{-0.5} \\
k_{clad}^2 \frac{C_5^2}{r_{c,0}^2} &= F^2 \cdot h_{1\phi}^2 \cdot (T_3(r_{c,o}) - T_f)^2 + S^2 \cdot h_{NB}^2 \cdot (T_3(r_{c,o}) - T_{sat})^2 \\
A_1 &= F^2 \cdot h_{1\phi}^2, A_2 = S^2 \cdot h_{NB}^2 \\
k_{clad}^2 \frac{C_5^2}{r_{c,0}^2} &= A_1 \cdot (T_3(r_{c,o})^2 - 2T_f T_3(r_{c,o}) + T_f^2) + A_2 \cdot (T_3(r_{c,o})^2 - 2T_{sat} T_3(r_{c,o}) + T_{sat}^2) \\
0 &= (A_1 + A_2) \cdot T_3(r_{c,o})^2 - 2(A_1 \cdot T_f + A_2 \cdot T_{sat}) T_3(r_{c,o}) + A_1 T_f^2 + A_2 T_{sat}^2 - k_{clad}^2 \frac{C_5^2}{r_{c,0}^2} \\
L_1 &= (A_1 + A_2), L_2 = -2(A_1 \cdot T_f + A_2 \cdot T_{sat}), L_3 = A_1 T_f^2 + A_2 T_{sat}^2 - k_{clad}^2 \frac{C_5^2}{r_{c,0}^2} \\
&\quad L_1 \cdot T_3(r_{c,0})^2 + L_2 T_3(r_{c,0}) + L_3 = 0 \\
T_3(r_{c,0}) &= \frac{-L_2 \pm \sqrt{L_2^2 - 4(L_1)(L_3)}}{2L_1} \\
T_3(r_{c,0}) - C_5 \ln r_{c,0} &= C_6
\end{aligned}$$

$$\begin{aligned}
& 5. T_2(r_{c,i}) = T_3(r_{c,i}) \\
C_3 \ln(r_{c,i}) + C_4 &= C_5 \ln(r_{c,i}) + C_6 \\
C_3 \ln(r_{c,i}) (\frac{k_{gap}}{k_{clad}} - 1) + C_6 &= C_4 \\
& 6. T_1(r_{fuel}) = T_2(r_{fuel}) \\
-\frac{q'(z)}{\pi r_{fuel}^2 \cdot k_{fuel}} \frac{r_{fuel}^2}{4} + C_2 &= C_3 \ln(r_{fuel}) + C_4 \\
C_2 &= C_3 \ln(r_{fuel}) + C_4 + \frac{q'(z)}{4\pi \cdot k_{fuel}}
\end{aligned}$$

$$C_1 = 0 \quad (3)$$

$$C_3 = \frac{-q'(z)}{2\pi \cdot k_{gap}} \quad (4)$$

$$C_5 = \frac{k_{gap}C_3}{k_{clad}} \quad (5)$$

$$C_6 = C_5 \left(\frac{-k_{clad}}{h \cdot r_{c,0}} - \ln(r_{c,o}) \right) + T_f \text{ (if } T_w < T_{sat} \text{)} \quad (6)$$

$$C_4 = C_3 \ln(r_{c,i}) \left(\frac{k_{gap}}{k_{clad}} - 1 \right) + C_6 \quad (7)$$

$$C_2 = C_3 \ln(r_{fuel}) + C_4 + \frac{q'(z)}{4\pi \cdot k_{fuel}} \quad (8)$$

4 Results

This first plot, Fig 1 shows the equilibrium quality throughout the channel for the fluid, with the step size decreasing allowing for finer lines and a more accurate solution. The final Δz of 0.01 m was chosen for all of the parameters calculated for this homework. Once it was selected, the converged solution for the equilibrium quality was given in Fig. 2.

The next measurements included temperatures in different parts of the system as a function of z ; Fig. 3 includes the center line temperature, wall and fluid temperature and the saturation temperature (based on the pressure).

This enlarged plot, in Fig. 4, shows the wall temperature crossing the saturation pressure. This means that there is 2ϕ heat transfer occurring without the presence of 2ϕ flow, since ($T_f \leq T_{sat}$) for the entire channel. 2ϕ heat transfer necessitates the use of a new correlation to calculate the temperature at the wall; the wall temperature remains continuous, but there is a change in slope based on the introduction of 2ϕ heat transfer.

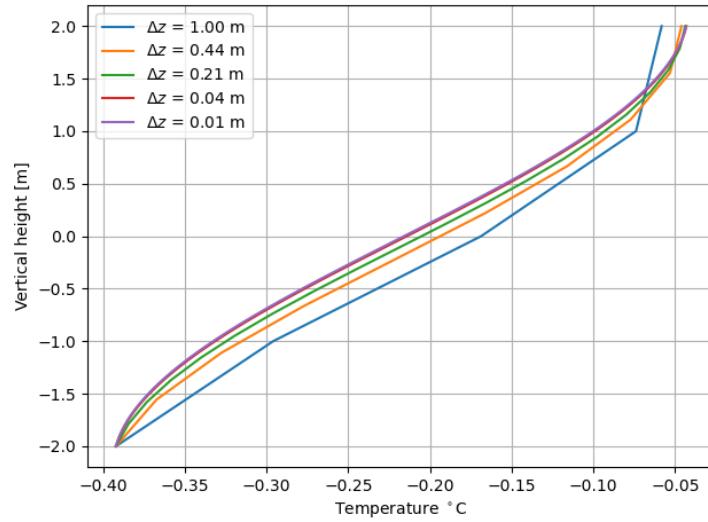


Fig. 1: χ_e convergence based on Δz where smaller step sizes are tested to find a good balance between computational intensity and accuracy

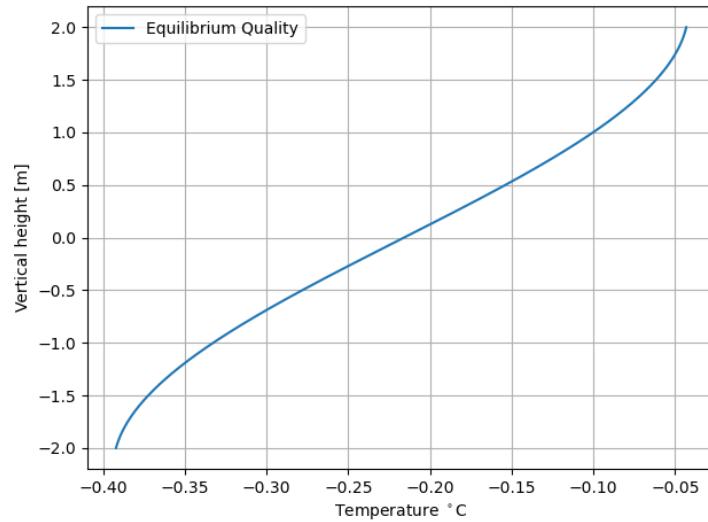


Fig. 2: χ_e , equilibrium quality, as the height in the channel increases

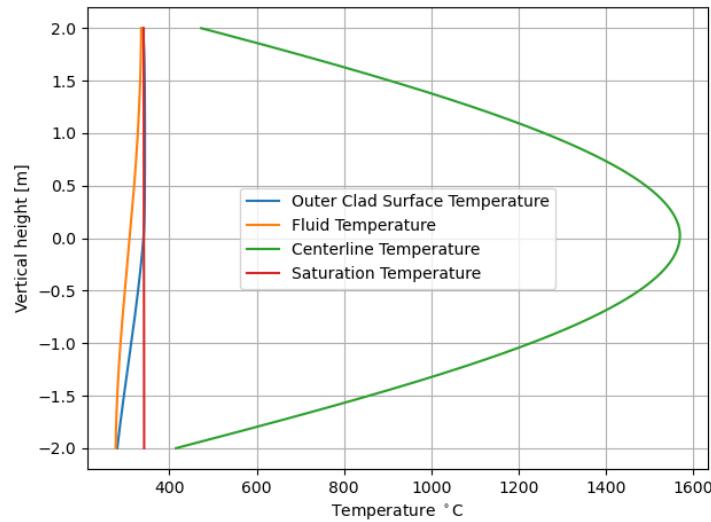


Fig. 3: Temperatures of different parts of the system as a function of the height of the channel, the overlap between fluid and wall is enlarged in Fig. 4

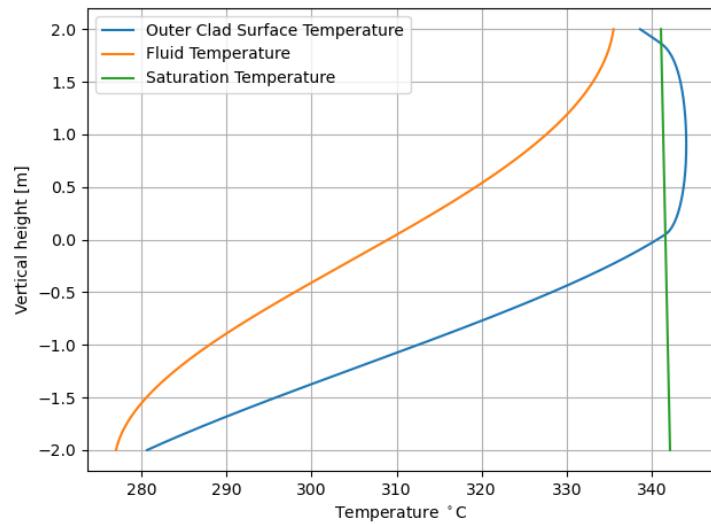


Fig. 4: Temperatures of fluid and wall as a function of the height of the channel

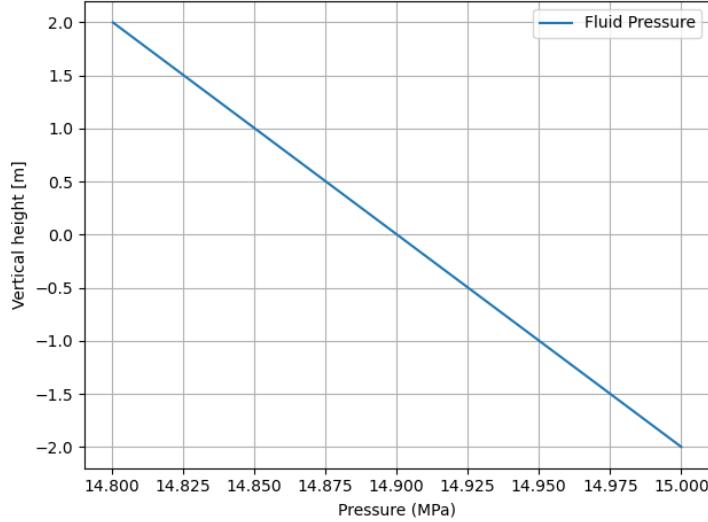


Fig. 5: Fluid pressure of the system as a function of the height of the channel, the range ends up being [14.8 - 15 MPa]

The final plot is pressure versus the change in height in the channel. This was done assuming constant properties for density throughout the channel and calculating the change in pressure pretty easily. Fig. 5 gives this relationship, assuming the density doesn't change from the inlet and neither does the friction factor (both used to calculate the change in pressure for a given Δz).

5 Code

Also available at Arnav Goyal's GitHub.

The plotting lines commented out at the bottom can be changed at the user's discretion to generate plots of their choosing.

```

import numpy as np
import matplotlib.pyplot as plt
from pyfluids import Fluid, FluidsList, Input

###variables###

H = 4 # m
He = 4.3 # m - extrapolated height
D_rod = 0.95 * 0.01 #m diameter
pitch = 1.26 * 0.01 #m distance between fuel pellets
D_fuel = 0.82 * 0.01#m diameter
gap_thick = 0.006 * 0.01 #m space between fuel and clad

```

```

k_gap = 0.25 # W / m degC
k_fuel = 3.6 # W / m degC
k_clad = 21.5 # W / m degC

r_fuel = D_fuel/2
r_clad_i = r_fuel + gap_thick
r_clad_o = D_rod/2

G = 4000 #kg/ m2 s or density times velocity
q0 = 380*100 # W/m linear initial heat generation
P_low = 15*(10**6) #MPa at z=-H/2
T_f_low = 277 #degC at z=-H/2
g = 9.8 #m/s2 acceleration due to gravity

wet_perim = np.pi * D_rod #m
area = pitch**2 - (np.pi*((D_rod/2)**2))
D_equiv = 4*area / wet_perim

MM_water = 18.02 #g/mol

water = Fluid(FluidsList.Water).unspecify_phase()
critP = water.critical_pressure

#####
#tempQualProp(temp, qual, prop):
def tempQualProp(temp, qual, prop):
    """
    Parameters
    -----
    temp: int
        Degrees celsius
    qual: int
        Number between 0 and 100 for flow quality
    prop: str
        one of the properties in the following list["vol", "intEnrg", "dynVisc", "enth", "rho"]
    """
    water_state = water.with_state(Input.temperature(temp), Input.quality(qual))
    if prop == "vol":
        return water_state.specific_volume #m3/kg
    elif prop == "intEnrg":
        return water_state.internal_energy #J/kg
    elif prop == "dynVisc":
        return water_state.dynamic_viscosity #Pa*s
    elif prop == "enth":
        return water_state.enthalpy #J/kg
    elif prop == "rho":
```

```

        return water_state.density #kg / m3
    elif prop == "pr":
        return water_state.prandtl
    elif prop == "k":
        return water_state.conductivity

def presQualProp(pres, qual, prop):
    """
    Parameters
    -----
    pres: int
        pressure in pascals
    qual: int
        Number between 0 and 100 for flow quality
    prop: str
        one of the properties in the following list["vol", "intEnrg", "dynVisc", "enth", "rho"]
    """
    water_state = water.with_state(Input.pressure(pres), Input.quality(qual))

    if prop == "vol":
        return water_state.specific_volume #m3/kg
    elif prop == "intEnrg":
        return water_state.internal_energy #J/kg
    elif prop == "dynVisc":
        return water_state.dynamic_viscosity #Pa*s
    elif prop == "enth":
        return water_state.enthalpy # J/kg
    elif prop == "rho":
        return water_state.density #kg / m3
    elif prop == "pr":
        return water_state.prandtl
    elif prop == "k":
        return water_state.conductivity

def tempPresProp(temp, pres, prop):
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        pressure in pascals
    temp: int
        Degrees celsius
    prop: str
        one of the properties in the following list["vol", "intEnrg", "dynVisc", "enth", "rho"]
    """

```

```

water_state = water.with_state(Input.temperature(temp), Input.pressure(pres))

if prop == "vol":
    return water_state.specific_volume #m3/kg
elif prop == "intEnrg":
    return water_state.internal_energy #J/kg
elif prop == "dynVisc":
    return water_state.dynamic_viscosity #Pa*s
elif prop == "enth":
    return water_state.enthalpy # J/kg
elif prop == "rho":
    return water_state.density #kg / m3
elif prop == "pr":
    return water_state.prandtl
elif prop == "k":
    return water_state.conductivity

def calchfg(temp=0, pres=0):
    if temp != 0:
        return tempQualProp(temp, 100, "enth") - tempQualProp(temp, 0, "enth") #J/kg
    else:
        return presQualProp(pres, 100, "enth") - presQualProp(pres, 0, "enth") #J/kg

def reynolds(visc):
    return (G*D_equiv/visc) # Unitless

def frictionfactor(re):
    return 0.316 * re**(-0.25)

def deltaP(rho, f, drho):
    var = 0.5*f*(G**2/rho)*(wet_perim/area) + rho*g
    if drho != 0:
        return -1*(var + (G**2)*drho)
    else:
        return -1*var

def calcXe(pres, temp):
    h = tempPresProp(pres=pres, temp=temp, prop="enth")
    hfg = calchfg(pres = pres)
    hf = presQualProp(pres=pres, qual=0, prop="enth")
    #print(h, hfg, hf, type(hf))
    return (h-hf)/hfg

def calcNewTemp(Xe, pressure):
    hf = presQualProp(pres=pressure, qual=0, prop="enth")
    hfg = calchfg(pres=pressure)

```

```

#print(Xe)
#print(Xe*hfg)
h = Xe*hfg + hf
#print(Xe, h, hf, hfg)

corresTemp = water.with_state(Input.enthalpy(h), Input.pressure(pressure)).temperature

return corresTemp, h # degrees Celsius

def dittusBoelter(k, re, pr):
    return 0.023 * re**(0.8) * pr**0.4 * (k/D_equiv)

def liuWinterton(S, F, htc, hnb, tf, tsat, tw):
    temporary = (F*htc*(tw-tf))**2 + (S*hnb*(tw-tsat))**2
    return np.sqrt(temporary)
#####
#####

option = 1 # 1 : PWR, 2: BWR

fig,ax = plt.subplots()

if option == 1 :

    for numOfPoints in [5, 10, 20, 100, 400]:
        #####
        Ps = [P_low]
        T_f_s = [T_f_low]
        Xes = [calcXe(P_low, T_f_low)]
        #print(Xes[0], calcNewTemp(Xes[0], Ps[0]))

        #print(f"P_i = {P_low}, T_f_i = {T_f_low}, Xe_i={Xes[0]}, hfg_i = {calchfg(pres=Ps[0])}")

        rhos = [tempPresProp(T_f_low, P_low, "rho")]
        mus = [tempPresProp(T_f_low, P_low, "dynVisc")]
        #print(f"rho_i = {rhos[0]}, mu_i = {mus[0]}")

        Res = [reynolds(mus[0])]
        frics = [frictionfactor(Res[0])]

        hs = [tempPresProp(T_f_low, P_low, "enth")]
        hfs = [presQualProp(P_low, 0, "enth")]
        hfgs = [calchfg(pres=P_low)]
        Prs = [tempPresProp(T_f_low, P_low, "pr")]

```

```

ks = [tempPresProp(T_f_low, P_low, "k")]
htcs = [dittusBoelter(ks[0], Res[0], Prs[0])]

Tsats = [water.dew_point_at_pressure(P_low).temperature]

#####
zs, deltaz = np.linspace(-H/2, H/2, numOfPoints, retstep=True)

qlins = q0*np.cos(np.pi*(zs/He)) # W / m
qdoubles = qlins/(np.pi*D_fuel)
#print(qlins[:10])

#####
c_3 = (-qlins[0])/(2*np.pi*k_gap)
c_5 = k_gap * c_3 / k_clad
c_6 = (c_5*((-k_clad/(htcs[0]*r_clad_o)) - np.log(r_clad_o))) + T_f_s[0]
c_4 = (c_3 *np.log(r_clad_i)*((k_gap/k_clad)-1))+c_6
c_2 = (c_3*np.log(r_fuel)) + c_4 + (qlins[0]/(4*np.pi*k_fuel))

T_f_c = [c_2]
T_c_in = [c_3*np.log(r_clad_i) + c_4]
T_c_out = [c_5*np.log(r_clad_o) + c_6]

#####

for i in range(1, len(zs)):
    #print(i)

    if i != 1 :
        drho = ((1/rhos[i-1])-(1/rhos[i-2])) * (1/(deltaz))
    else:
        drho = 0

    #print(f"drho = {drho}")

deltaPCurrent = deltaP(rhos[0], frics[0], 0) #delta P / delta z using constant p
#print(f"delta P = {deltaPCurrent}")

newP = deltaPCurrent*deltaz + Ps[i-1]
#print(f"P = {newP}")
Ps.append(newP)

```

```

newhfg = calchfg(pres=Ps[i])
newhf = presQualProp(Ps[i], 0, "enth")
#print(f"newhfg = {newhfg}")
qdouble = qdoubles[i]

#print(qdouble, qdouble*wet_perim/G*area)
newXe = (((qdouble*wet_perim*deltaz)/(area*G))-(newhf-hfs[i-1])-(Xes[i-1]*(newhf-hfs[i])))

#print(f"hfg = {newhfg}, Xe = {newXe}")
Xes.append(newXe)
hfgs.append(newhfg)
hfs.append(newhf)

newT_f, newh = calcNewTemp(Xes[i], pressure=Ps[i])
T_f_s.append(newT_f)
Tsats.append(water.dew_point_at_pressure(Ps[i]).temperature)
hs.append(newh)

rhos.append(tempPresProp(pres=Ps[i], temp=T_f_s[i], prop="rho"))
#print(rhos[i])
mus.append(tempPresProp(pres=Ps[i], temp=T_f_s[i], prop="dynVisc"))
Res.append(reynolds(mus[i]))
frics.append(frictionfactor(Res[i]))
ks.append(tempPresProp(pres=Ps[i], temp=T_f_s[i], prop="k"))
Prs.append(tempPresProp(pres=Ps[i], temp=T_f_s[i], prop="pr"))
htcs.append(dittusBoelter(ks[i], Res[i], Prs[i]))

#print(f"P_{i} = {Ps[i]}, T_f_{i} = {T_f_s[i]}, Xe_{i}={Xes[i]}")
#print(f"rho_{i} = {rhos[i]}, mu_{i} = {mus[i]}")
if Xes[i] < 0:
    chi = 0
elif Xes[i] > 1:
    chi = 1
else:
    chi = Xes[i]
#####
#####Temperature inside the fuel rod#####

c_3 = (-qlins[i])/(2*np.pi*k_gap)
c_5 = k_gap * c_3 / k_clad
if T_c_out[i-1] <= Tsats[i-1]:
    c_6 = (c_5*((-k_clad/(htcs[i]*r_clad_o)) - np.log(r_clad_o))) + T_f_s[i]
else:

```

```

    curP = Ps[i]
    F = (1+(chi*Prs[i]*((presQualProp(curP, 0, "rho")/presQualProp(curP, 100, "rho"))))**(-1))
    S = (1+(0.055*(F**0.1)*(Res[0]**0.16)))**(-1)
    hnb = 55*((curP/critP)**0.12)*(qdoubles[i]**(2/3))*((-1*np.log10(curP/critP)))
    '''tw = Tsats[i]
    #print(S, F, hnb, tw, T_f_s[i], Tsats[i], liuWinterton(S, F, htcs[i], hnb, T_f_c[i], Tsats[i], tw)

    while (qdoubles[i] - liuWinterton(S, F, htcs[i], hnb, T_f_s[i], Tsats[i], tw) >= 1e-05):
        tw = tw*1.001'''

    A1 = (F**2) * (htcs[i]**2)
    A2 = (S**2) * (hnb**2) #If F and S are flipped there is a discontinuity in the code
    L1 = A1+A2
    L2 = -2*(A1*T_f_s[i] + A2*Tsats[i])
    L3 = A1*(T_f_s[i]**2) + A2*(Tsats[i]**2) - ((k_clad*c_5/r_clad_o)**2)
    #print(f"F = {F}, S = {S:0.2f}, hnb={hnb:0.2f}, A1 = {A1:0.2f}, A2 = {A2:0.2f}, L1 = {L1:0.2f}, L2 = {L2:0.2f}, L3 = {L3:0.2f}")
    #T_w_min = ((-L2-((L2**2 - (4*L1*L3))**0.5))/(2*L1))
    T_w_plus = ((-L2+((L2**2 - (4*L1*L3))**0.5))/(2*L1))
    #print("Wall temp", T_w_plus)
    c_6 = T_w_plus - (c_5*np.log(r_clad_o))

c_4 = (c_3 *np.log(r_clad_i)*((k_gap/k_clad)-1))+c_6
c_2 = (c_3*np.log(r_fuel)) + c_4 + (qlins[i]/(4*np.pi*k_fuel))

T_f_c.append(c_2)
T_c_in.append(c_3*np.log(r_clad_i) + c_4)
T_c_out.append(c_5*np.log(r_clad_o) + c_6)
#print(c_5*np.log(r_clad_o) + c_6, qdoubles[i])

#ax.plot(qlins, zs, label="q\'")
#ax.plot(np.array(Ps)/(10**6), zs, label="Fluid Pressure")

#ax.plot(T_c_out, zs, label="Outer Clad Surface Temperature")
#ax.plot(T_f_s, zs, label="Fluid Temperature")
#ax.plot(T_f_c, zs, label="Centerline Temperature")
#ax.plot(Tsats, zs, label="Saturation Temperature")
# ax.plot(T_c_in, zs, label="Inner Clad Surface Temperature")

ax.plot(Xes, zs, label=f"\Delta z = {deltaz:0.2f} m")
#ax.plot(rhos, zs, label="Densities")

```

```
#ax.plot(mus, zs, label="Dynamic Viscosities")
#ax.plot(Res, zs, label="reynolds")
#ax.plot(frics, zs, label = "frics")

#ax.set_xscale('log')
ax.set_ylabel(f"Vertical height [m]")
ax.set_xlabel(f"Temperature  ${}^{\circ}\text{C}$ ")
ax.grid()
ax.legend()
plt.tight_layout()

plt.show()
```