

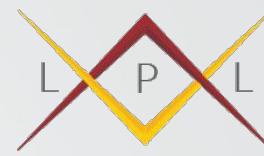
# ATLAS CDR

Liquid Propulsion Laboratory

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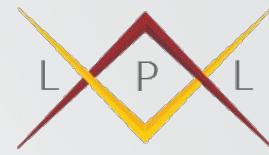
- LPL currently has a lack of experience designing around and working with cryogenic liquids and cryogenic operations
  - We are looking to the Alumni for help to better understand industry cryogenic design considerations and standards for operations
- Please provide input and ask questions regarding any ConOps or design decisions we address in this CDR, and please provide us with advice regarding anything we did not address, or will likely encounter in the future.



# Agenda



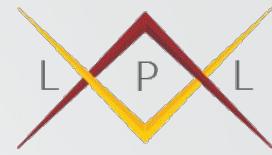
1. Introduction & Goals
  - a. CDR Entrance and Exit Criterion
  - b. Project Planning
2. Feed System
  - a. Updated P&ID
  - b. CAD Overview
  - c. FMEA
3. Feed System Boil-Off and Vent Sizing Calculations
4. Feed System Specifications
  - a. Component Specifications
  - b. Pressure Drop Calculations
  - c. Cost
5. Engine Mount
6. Testing Campaign
7. Summary & Questions



## Purpose: Review of Atlas test stand and subsystems design

The following conditions have been met which enable a CDR of the system:

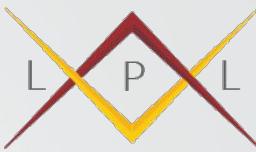
- Feed system P&ID final revision completed and approved
- Feed system lines, pressurant, propellant, and regulator specifications confirmed
- Feed system setup and hot fire ConOps
- Feed system FMEA (Failure Mode and Effects Analysis)
- Feed system final cost
- Complete CAD



The following conditions will allow Atlas to exit CDR and enter the manufacturing stage:

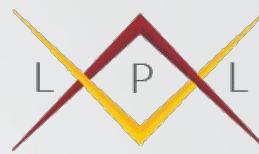
- Determine that a satisfactory basis has been achieved for preceding to a finalized design and test procedures
- Confirmation of sufficient funding from department

# System Requirements



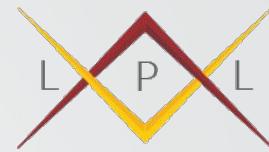
- Feed System shall be capable of remote safing in the event of a hazard to personnel
- Feed System shall be single fault tolerant against critical hazards during first hot-fire
- Feed System shall be transportable to testing locations
- Feed System components shall be easily accessible for integration, inspection, and calibration once assembled
- Liquid Oxygen flow velocity shall be below  $30 \text{ m/s}$  to minimize risk of impact ignition, per NASA Safety Standard for Oxygen and Oxygen System (NASA 1740.15)

# System Requirements Cont.

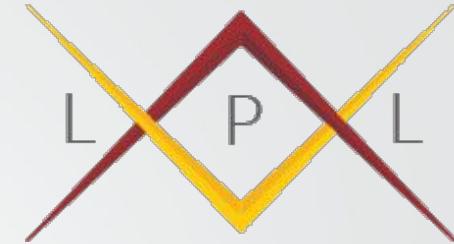


- Feed System shall incorporate independently powered vent valves, per FAR's REDS Standards
- Feed System shall reuse as many components as possible from the Original Hydra Feed System
- The Feed System shall support the flow and pressure requirements of the Balerion-Dev2 Engine
- The Feed System may support any engine LPL may develop in the future

# Project Planning



- Start LOx and Pressurant Side building as soon as CDR is done
  - Aiming to have LOx side built by July 15th
- **DAQ CDR date: July TBD**
- Aiming to have Fuel Side built by August 15th
- Testing Campaign starting on August 16th
- Hot fire by Aerospace Corporation SOW completion date (December)



# Atlas

# Feed System

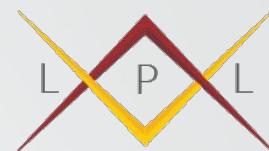
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# P&ID Component Directory



**Components Directory**

Pneumatic Ball Valve	Pressure Relief Valve	Rupture Disk	Pressure Vessel
Hand Ball Valve	Solenoid Valve	Pump	Quick Disconnect One-Way Checking
Check Valve	Pressure Regulator	Flowmeter	Quick Disconnect Two-Way Checking
Needle Valve	Filter		Quick Disconnect Non-Checking

**Line Colors:**

Nitrogen → Oxygen → Kerosene → Air → Flex Hose

**Number Scheme:**

AAA: Acronym  
F: Fluid Number  
##: Tag Number

**Symbols:**

**AAA\_Instrument**

**AAA\_Instrument**

**AAA\_Instrument/Acronym**

**F\_Fluid**

<b>Number Scheme:</b>    <b>AAA: Acronym</b> <b>F: Fluid Number</b> <b>##: Tag Number</b>	<b>Symbols:</b>  <b>AAA</b> <b>F##</b>	<b>AAA_Instrument</b>  <b>PT</b> Pressure Transmitter <b>PG</b> Pressure Gauge <b>TT</b> Temperature Transmitter <b>FM</b> Flowmeter <b>CYL</b> Gas Cylinder <b>CKV</b> Check Valve <b>FLTR</b> Filter	<b>AAA_Instrument</b>  <b>HBV</b> Hand Ball Valve <b>PBV</b> Pneumatic Ball Valve <b>PMP</b> Pump <b>PRV</b> Pressure Relief Valve <b>EPR</b> Electronic Pressure Regulator <b>RD</b> Rupture Disk (Burst Disk) <b>SV</b> Solenoid Valve <b>NV</b> Needle Valve <b>TK</b> Tank	<b>AAA_Instrument/Acronym</b>  <b>V</b> Pressure Vessel <b>FH</b> Flex Hose <b>NC</b> Normally Closed <b>NO</b> Normally Open <b>BT</b> Bottle <b>QD</b> Quick Disconnect
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**AAA: Acronym**  
**F: Fluid Number**  
**##: Tag Number**

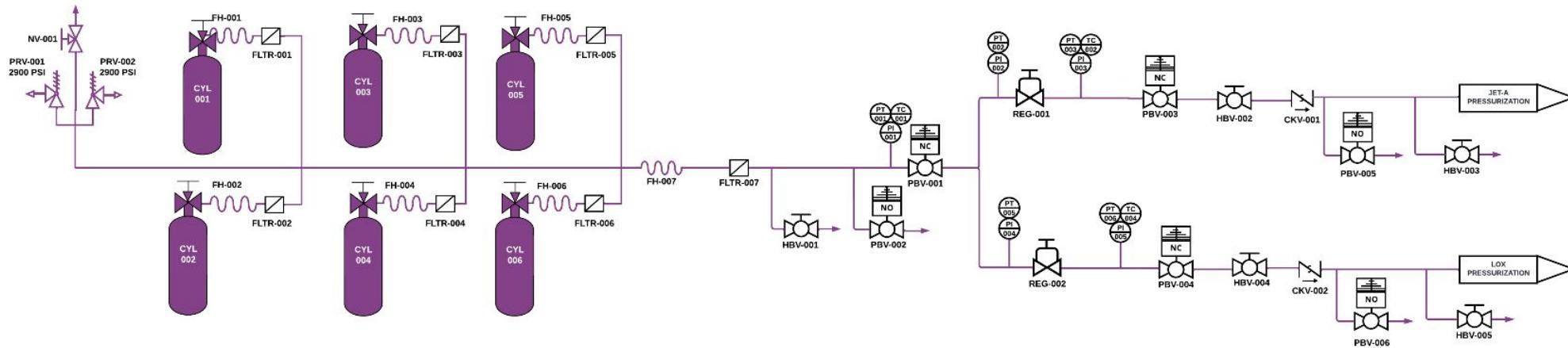
**PT** Pressure Transmitter  
**PG** Pressure Gauge  
**TT** Temperature Transmitter  
**FM** Flowmeter  
**CYL** Gas Cylinder  
**CKV** Check Valve  
**FLTR** Filter

**HBV** Hand Ball Valve  
**PBV** Pneumatic Ball Valve  
**PMP** Pump  
**PRV** Pressure Relief Valve  
**EPR** Electronic Pressure Regulator  
**RD** Rupture Disk (Burst Disk)  
**SV** Solenoid Valve  
**NV** Needle Valve  
**TK** Tank

# Feed System Overview - Pressurant P&ID

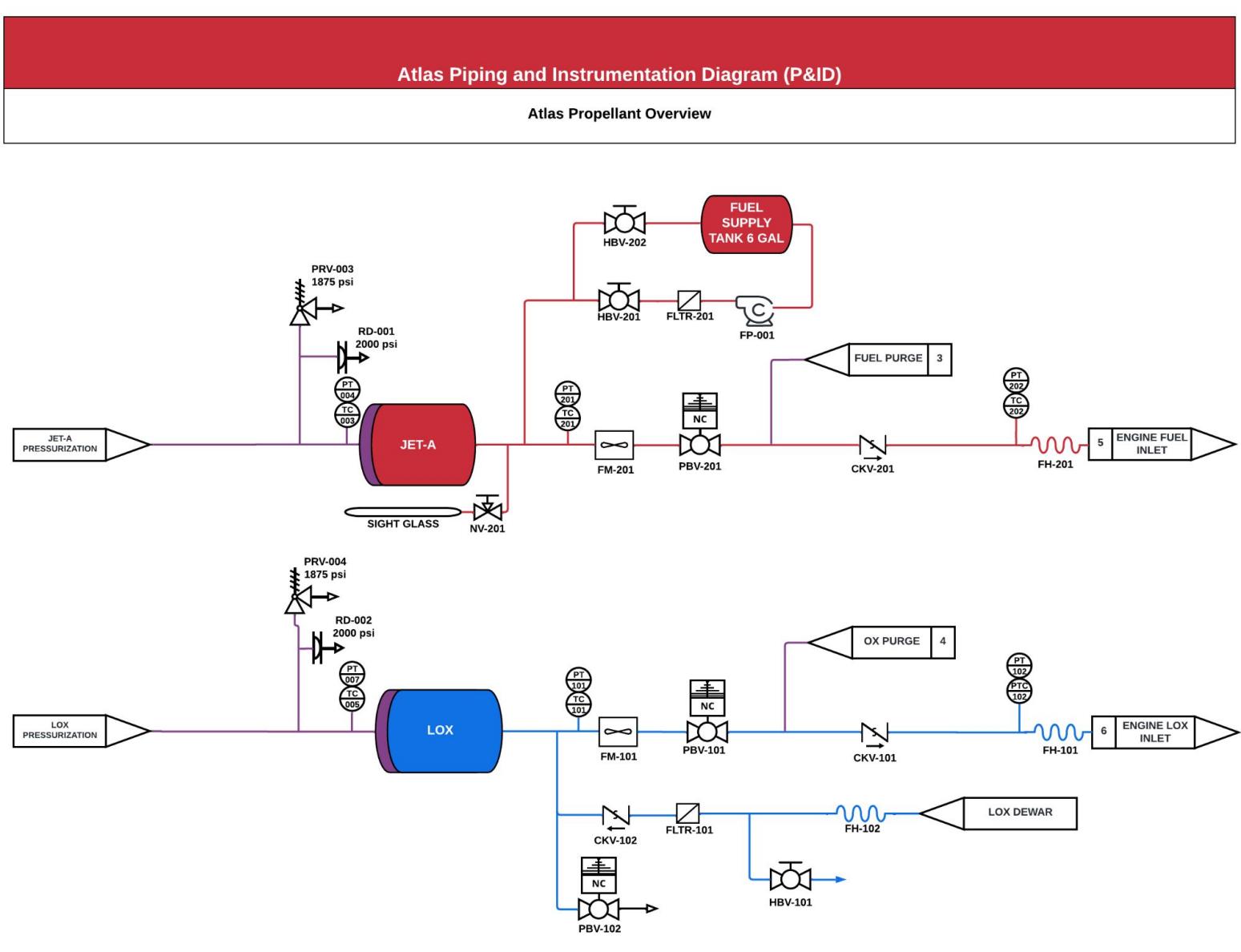
Atlas Piping and Instrumentation Diagram (P&ID)

MNMS & Atlas Pressurant Overview



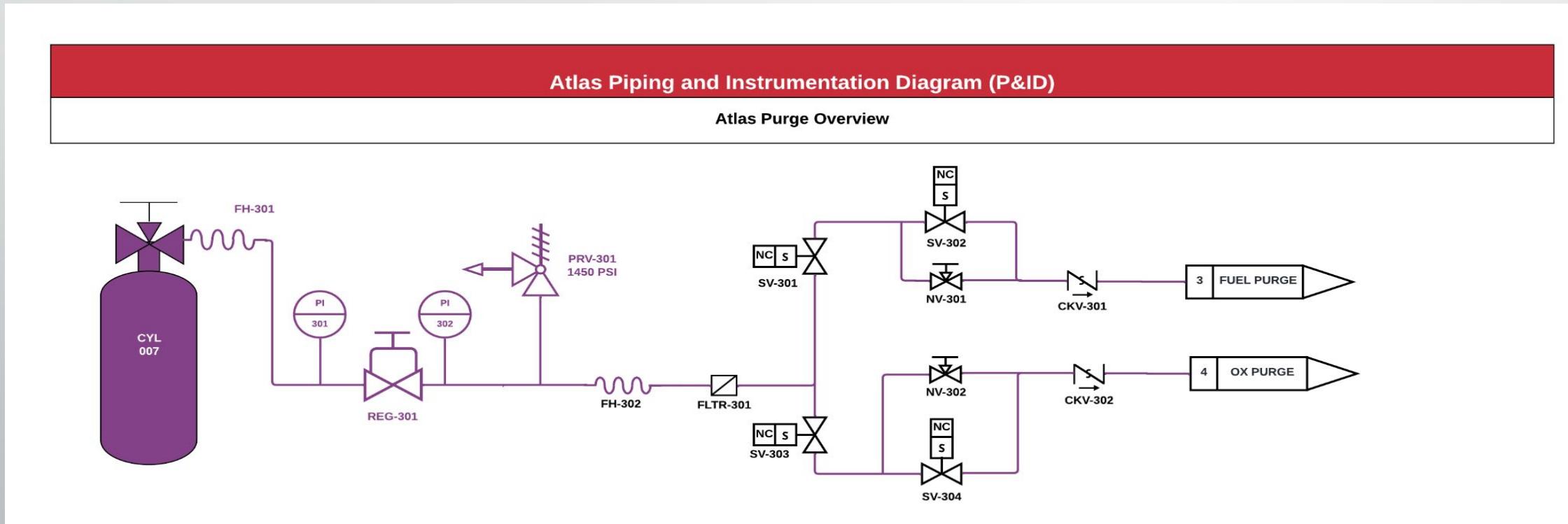
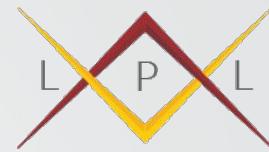
- GN2 Pressurant gas cylinders will be 49L at 2700 psig
  - Port implemented for future remote fill
  - HBV-001, PBV-001 & PBV-002 will be High Pressure Valves
  - PBV-001 will be actuated slowly to reduce water hammer on the regulators
- Filters on all bottles will limit risk of contamination
- HBV-001, HBV-002 & HBV-003 are manually operated in the event the system must be depressed manually (FAR recommendation)

# Feed System Overview - Propellants



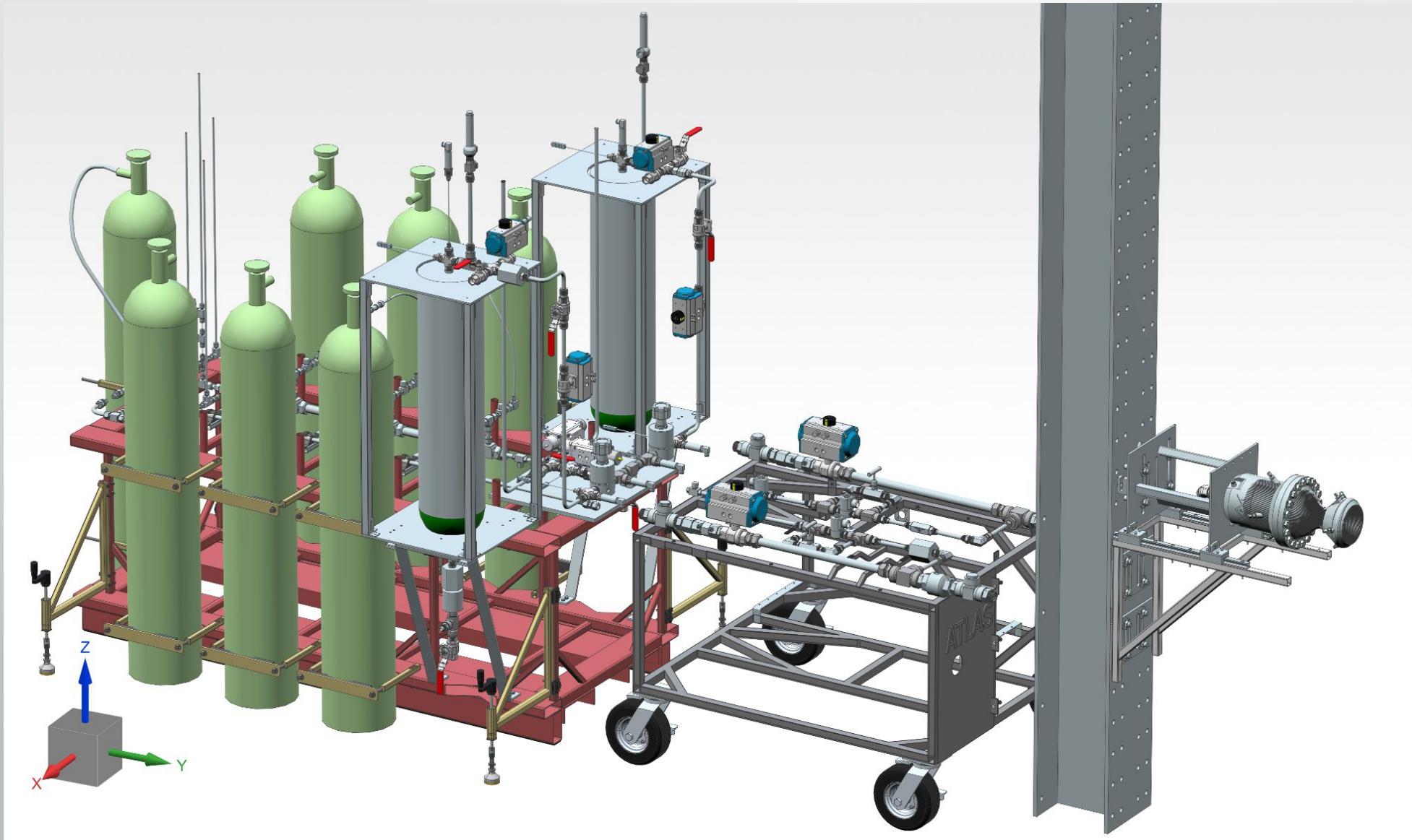
- Fuel will be filled and drained into a 6 Gal non-pressurized tank
- Fuel tank level will be determined with a sight glass
- LOx filled by dewar and drained into ambient
  - FAR allows for LOx to be drained on ground
- LOx tank level will be determined with a manometer
- Filters downstream of LOx and Fuel inlets to limit risk of contamination

# Feed System Overview - Purge

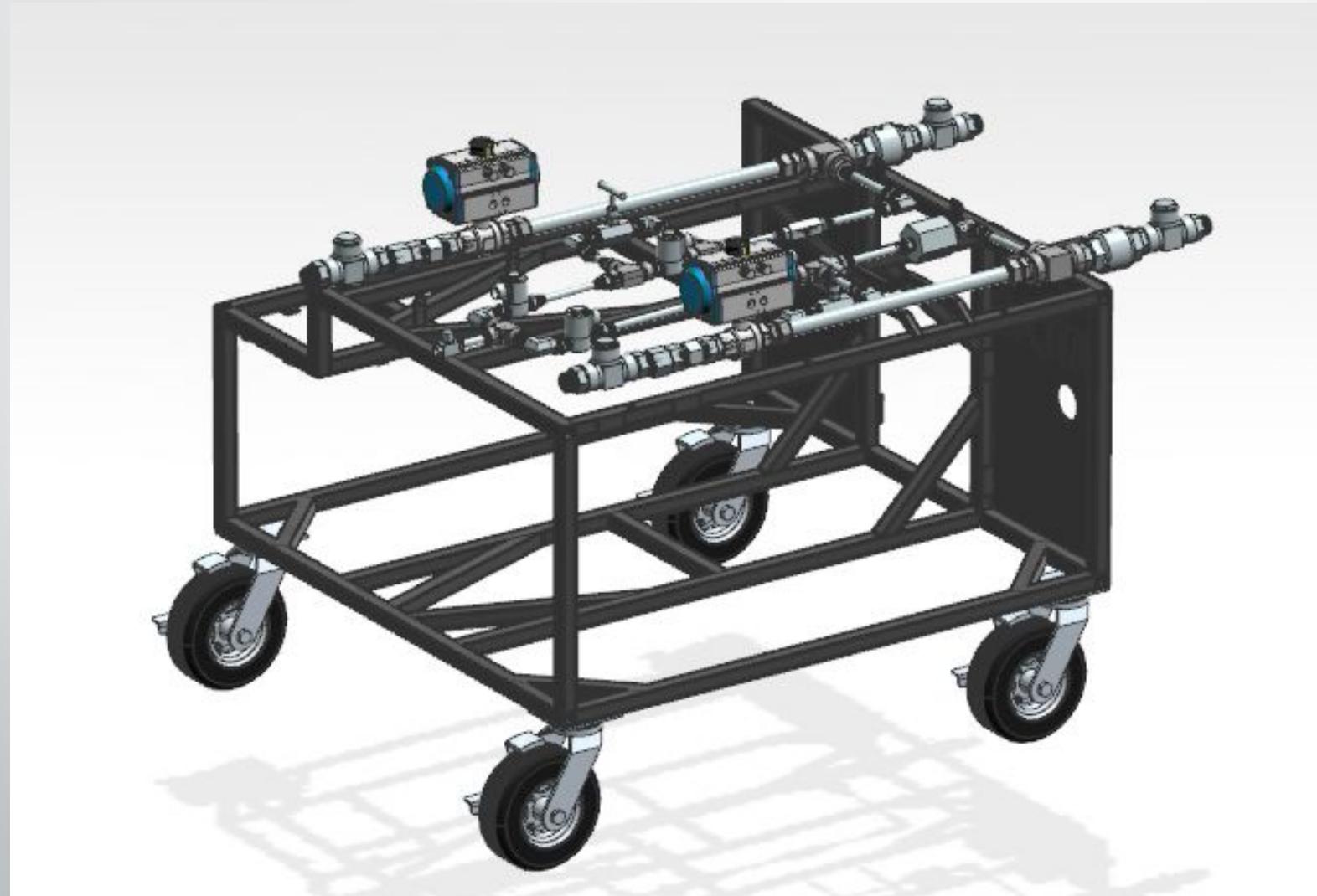


- Will be used for both trickle purge and post-fire purge
  - More about this in ConOps
- CKV-302 will need to be cryo & oxygen rated

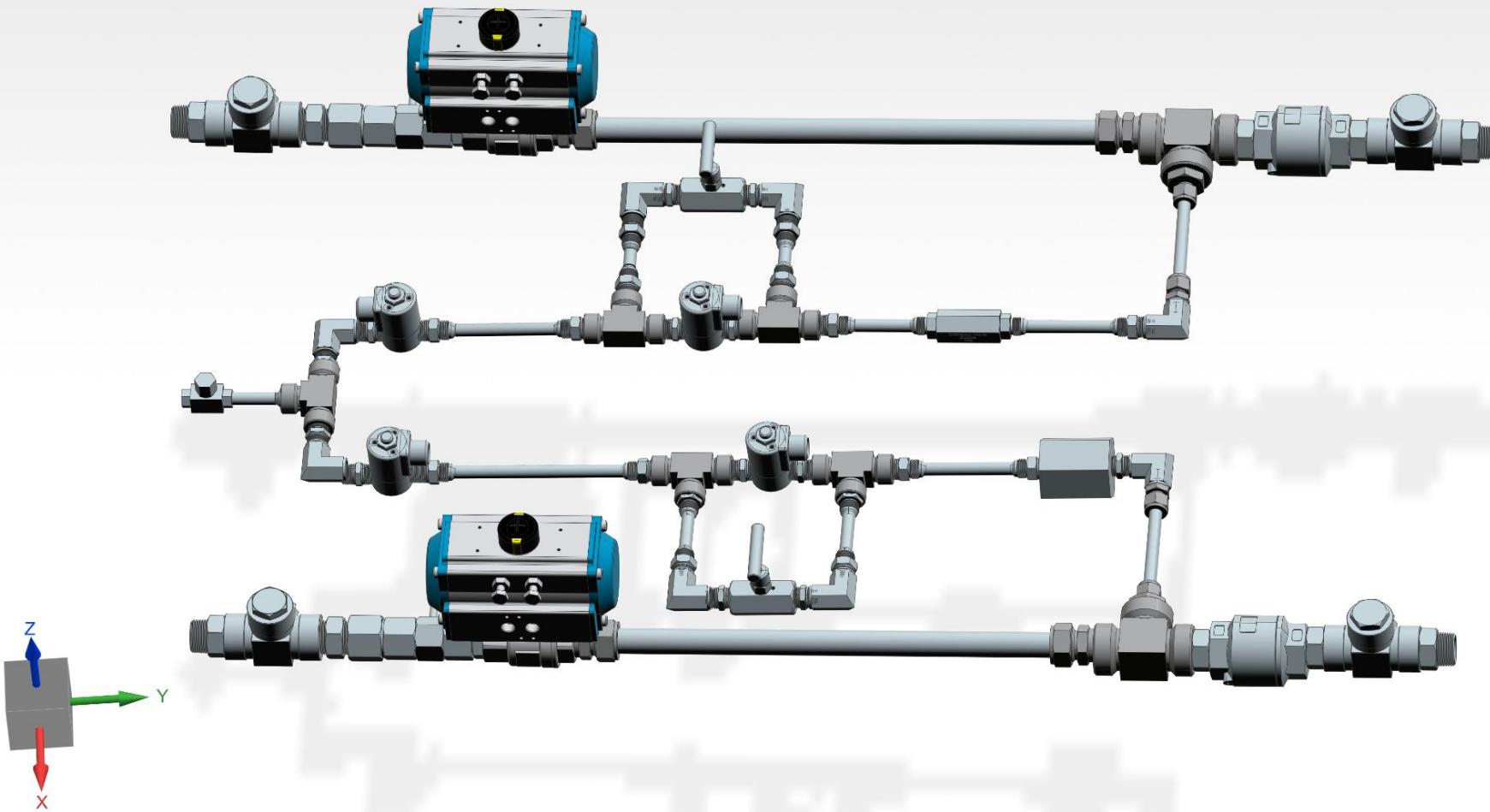
# CAD Overview



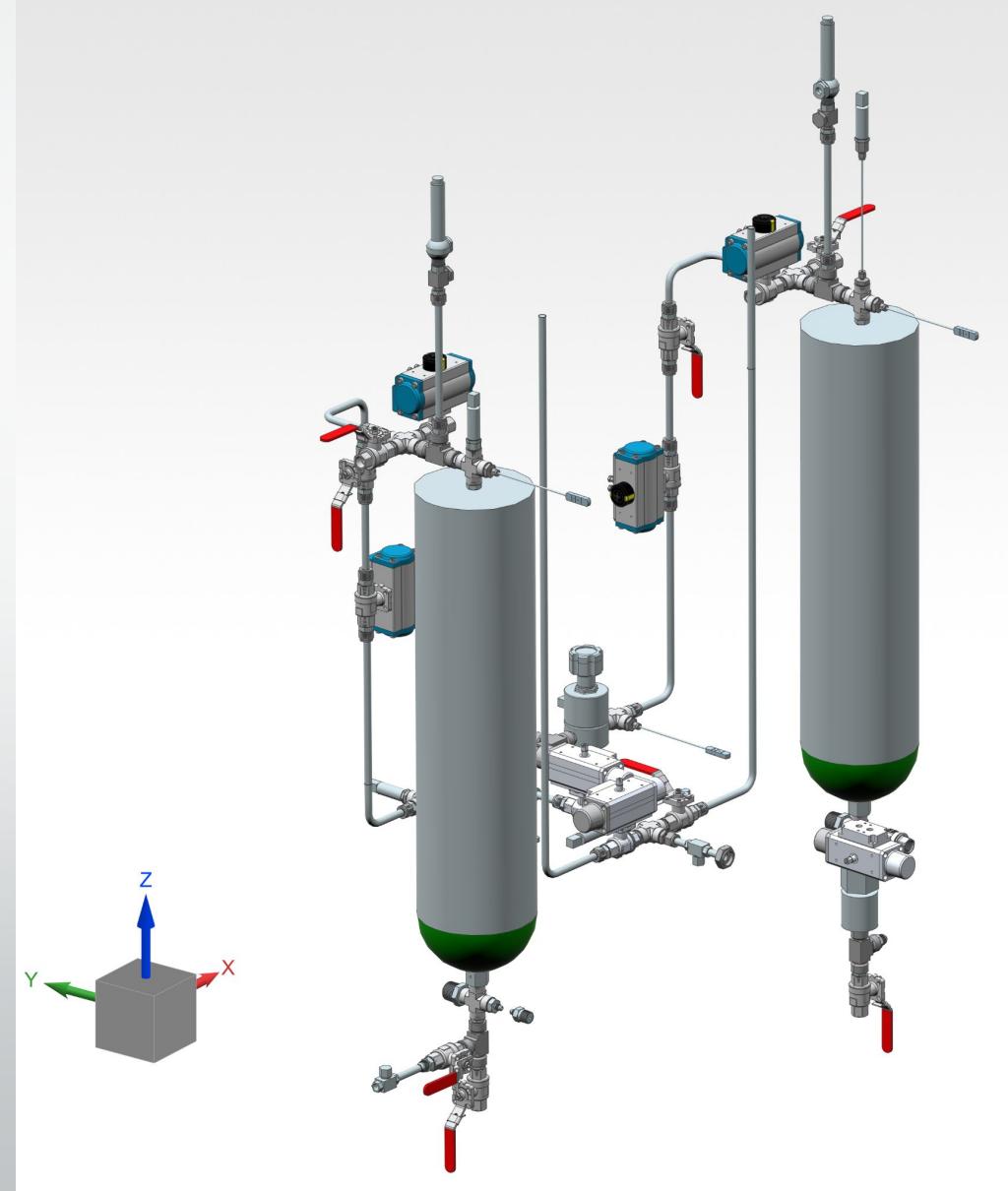
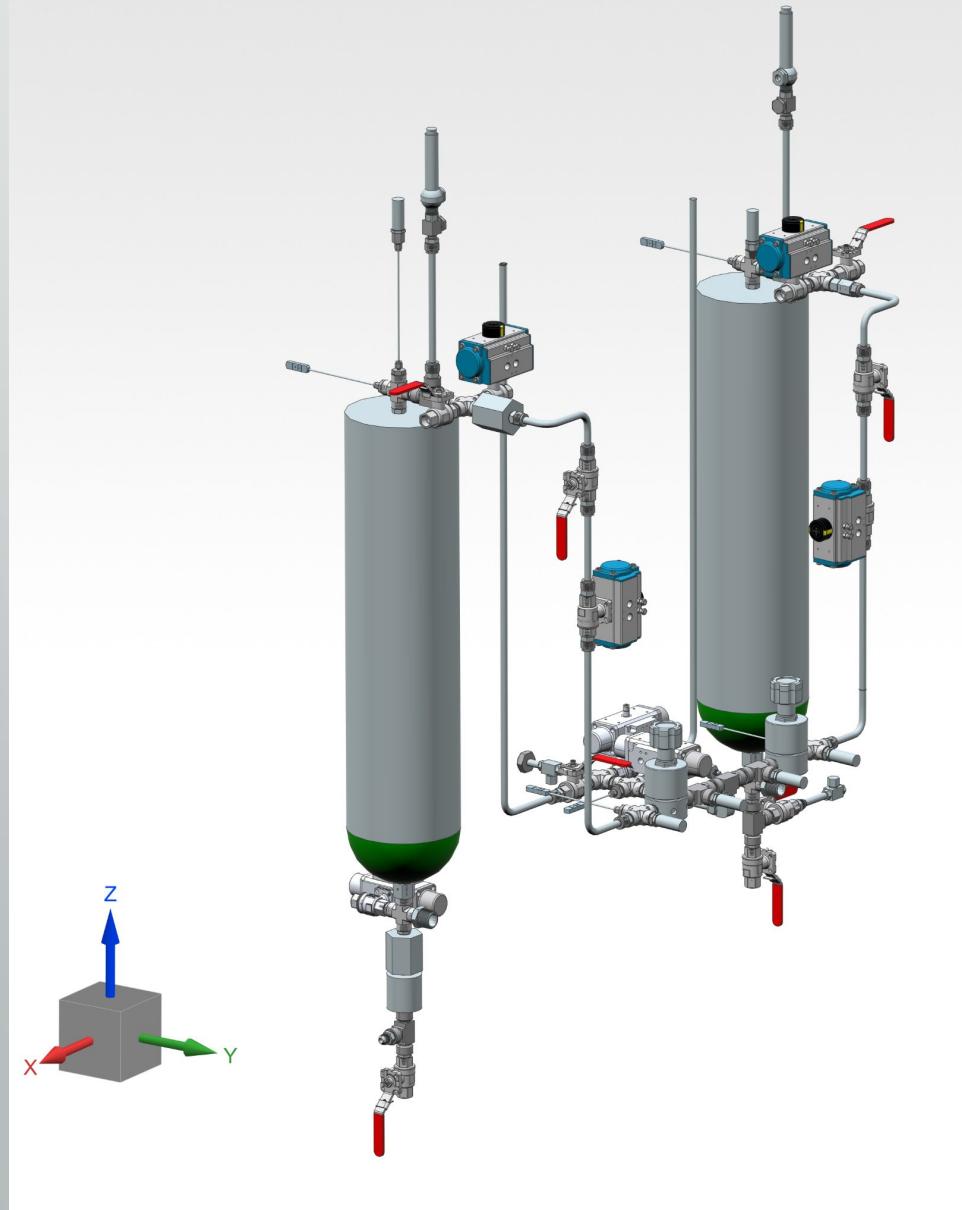
# CAD Overview: Atlas



# CAD Overview: Fuel, LOx, Purge Lines



# CAD Overview: Fuel and LOx Tank Interface



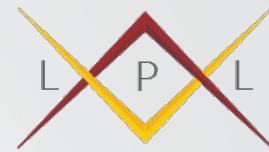
# FMEA - Hazard Likelihoods and Severities



Failure Mode Likelihood	Probability Value	Failure Mode Likelihood Criterion
Rare	1	Failure mode not expected to occur, or design controls implemented to eliminate the failure mode. (0-20%)
Unlikely	2	Overall low chance of occurring, or little to no past occurrence of the failure mode. (20-40%)
Possible	3	Overall probability of failure to occur is unlikely, but the failure mode is inherent to the project if not eliminated by design. (40-60%)
Likely	4	History of several occurrences of the failure mode in heritage projects, or failure mode has never occurred, but conditions present for the failure mode to occur. (60-80%)
Very Likely	5	Failure mode is expected to occur, higher than normal chance that controls implemented may not stop the hazard. (80-100%)

Failure Mode Severity	Probability Value	Failure Mode Severity Criterion
Trivial	1	Damage to hardware, facilities, or equipment akin to normal wear and tear, no injury to personnel, project progress not impacted, cost overrun of < \$30.
Minor	2	Damage to facilities, equipment, or test stand hardware more than normal wear and tear level, minor injury to personnel, progress slightly impacted but does not impact internal development milestones, cost overrun of < \$500.
Moderate	3	Some damage to property or facilities, moderate injury to personnel requiring medical attention, internal schedule slip that does not result in Aerospace Corp. SOW completion date slip, cost overrun of < \$1,000.
Critical	4	Risk results in significant injury, significant damage to property, facilities, systems, equipment, test stand hardware, progress temporarily halted, test stand requirement not met, cost overrun of < \$10,000.
Catastrophic	5	Risk results in life-threatening injury or death, risk causes loss of multiple major test stand components or facility destruction, results in project failure, cost overrun of > \$10,000.

# FMEA - Risk Assessment Matrix



Likelihood	Severity				
	Negligible (1)	Minor (2)	Moderate (3)	Major (4)	Catastrophic (5)
Rare	2	3	4	5	6
Unlikely	3	4	5	6	7
Possible	4	5	6	7	8
Likely	5	6	7	8	9
Very Likely	6	7	8	9	10
Risk Level	Low (2-4)	Medium (5-6)	High (7-8)	(Critical 9-10)	

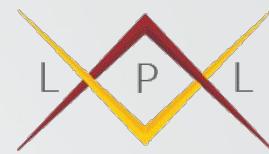
## Definitions:

- IRA: Initial Risk Assessment
- PMRA: Post-Mitigation Risk Assessment

## Control Categories Implemented:

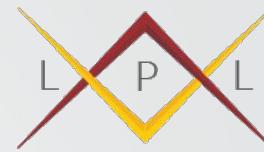
- Detective : Detective controls are designed to detect errors or irregularities that may have occurred.
- Preventative : Preventive controls are designed to keep errors and irregularities from occurring in the first place.
- Corrective: Corrective controls are designed to correct errors or irregularities that have been detected.

# FMEA - Top 5 Failure Modes

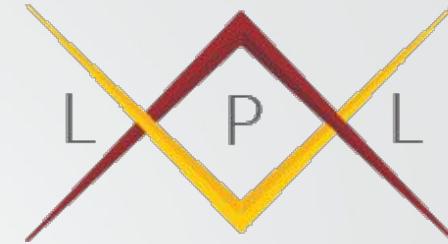


Risk Title	IRA	Mitigations	PMRA
High Pressure in LOx Tank	9 Catastrophic Likely	<p><b>Preventative:</b></p> <ul style="list-style-type: none"> <li>- 1.5x MEOP (per SMC-s-016) Hydrostatic Test of LOx tank.</li> </ul> <p><b>Detective:</b></p> <ul style="list-style-type: none"> <li>- PT readings of tank pressure to determine if the system has or will reach the MEOP.</li> </ul> <p><b>Corrective:</b></p> <ul style="list-style-type: none"> <li>- PRV-004 &amp; PBV-006, HBV-005 &amp; RD-002 rupture to release pressure, no longer able to operate the system unless RD is replaced.</li> </ul>	7 Catastrophic Unlikely
High Temperature in LOx Tank	9 Catastrophic Likely	<p><b>Preventative:</b></p> <ul style="list-style-type: none"> <li>- Ensure adequate tank insulation using foam then foil.</li> <li>- Wrap Reflective Shielding (foil blanket) around LOx tank to reflect sunlight.</li> <li>- Determine testing day with adequate weather conditions where the temperature does not exceed 100 degrees F.</li> </ul> <p><b>Detective:</b></p> <ul style="list-style-type: none"> <li>- PT readings of tank pressure to determine if the system has or will reach the MEOP.</li> </ul> <p><b>Corrective:</b></p> <ul style="list-style-type: none"> <li>- PRV-004 &amp; PBV-006, HBV-005 &amp; RD-002 rupture to release pressure, no longer able to operate the system unless RD is replaced.</li> </ul>	7 Catastrophic Unlikely

# FMEA - Top 5 Failure Modes



Risk Title	IRA	Mitigations	PMRA
Contamination in LOx Tank	9 Catastrophic Likely	<p><b>Preventative:</b></p> <ul style="list-style-type: none"> <li>- Clean tank per the LOx cleaning procedures (per NASA 1740.15).</li> <li>- Purge all lines and tanks before and after testing operations.</li> <li>- Conducted trade study in order to determine the best FLTR based on criteria including MEOP, cost, quality, CV, lab experience, etc.</li> </ul>	6 Catastrophic Rare
High Pressure in Kerosene Tank	8 Catastrophic Possible	<p><b>Preventative:</b></p> <ul style="list-style-type: none"> <li>- 1.5x MEOP (per SMC-s-016) Hydrostatic Test of fuel tank.</li> </ul> <p><b>Detective:</b></p> <ul style="list-style-type: none"> <li>- PT readings of tank pressure to determine if the system has or will reach the MEOP.</li> </ul> <p><b>Corrective:</b></p> <ul style="list-style-type: none"> <li>- PBV-005, HBV-003, PRV-003, and RD-001 to release pressure, no longer able to operate the system unless RD is replaced.</li> </ul>	6 Catastrophic Rare
PRV-004 on LOx side fails	8 Catastrophic Possible	<p><b>Preventative:</b></p> <ul style="list-style-type: none"> <li>- Conducted trade study in order to determine the best PRV based on criteria including MEOP, cost, quality, CV, lab experience, etc.</li> </ul> <p><b>Corrective:</b></p> <ul style="list-style-type: none"> <li>- Able to vent through RD-002 or HBV-005 as last resort since PBV-006 would have already failed.</li> </ul>	6 Major Unlikely



# Feed System Boil-Off and Vent Sizing Calculations

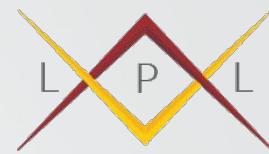
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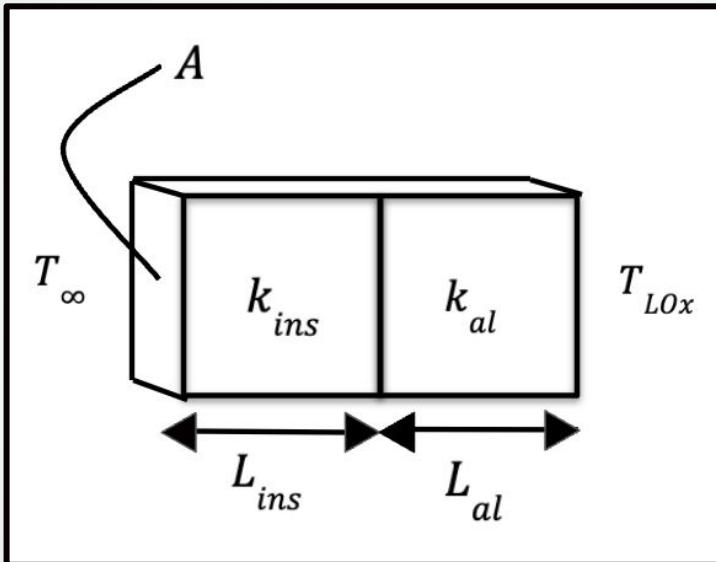


# Boil-Off Calculations



## Heat Transfer Equation (Two Layers)

$$\dot{Q} = \frac{T_{\infty} - T_{Lox}}{\frac{L_{al}}{k_{al} A} + \frac{L_{ins}}{k_{ins} A}}$$



A: Area  
k: Thermal Conductivity  
L: thickness  
ins: foam insulator  
al: aluminum tank

Divide Heat Flux by Enthalpy of Vaporization to get boil off mass flow rate



$$\dot{m} = \frac{\dot{Q}}{\Delta h_{vap}}$$

# Boil-Off Calculations

## Boundary Conditions:

$T_{\infty} = 180 \text{ }^{\circ}\text{F}$  (hot day, direct sunlight)

$T_{\text{LOx}} = -297.3 \text{ }^{\circ}\text{F}$  (boiling temp. Ox)

## Aluminum:

$$k_{\text{al}} = 237 \text{ [W/(m*K)]}$$

$$L_{\text{al}} = 0.391 \text{ [in]}$$

## Foam Insulator:

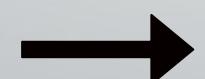
$$k_{\text{ins}} = 0.039 \text{ [W/(m*K)]}$$

$$L_{\text{ins}} = 0.25 \text{ [in]}$$

## Final Value

$$\dot{m} = 0.0042 \frac{\text{kg}}{\text{s}}$$

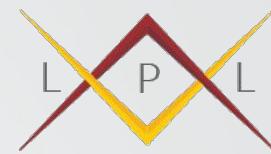
Convert to SCFM



6.22 SCFM



# Vent Sizing Calculations



**Boil-Off + Fill Rate,**  
converted from SCFM to GPM

$$C_v \geq Q \sqrt{\frac{SG}{\Delta p}}$$

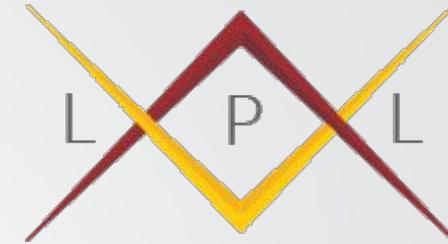
Specific Gravity of **cold GOx**



$$C_{v,vent} \geq 6.41$$

( $C_v$  of vent must be of at least this value to ensure volumetric flow rate of boil off + fill rate can be sustained)

$\Delta p$ : size vent for steady-state venting at low pressure differential (**1-2 psi**) to prevent high pressure build-up



# Feed System Components & Specifications

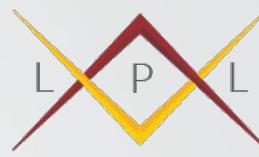
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# Motivation



- We wanted to reuse as many valves, fittings, and components from Hydra as possible.
  - Despite new funding for Atlas, the original objective of recycling as much as possible remains

# Check Valve Selection



**Feed System Side:** Purge, Pressurant

**P&ID Reference:** CKV 302, CKV 001

**Vendor:** Check-All

**Part Number:** SS-8CP2-1

**Cv:** 4.3

**Inlet/Outlet:**  $\frac{1}{2}$  in. Female NPT

**Max Pressure:** 3000 psig

**\*Reusing from Hydra (formerly CVK-003, CVK-002)**



**Feed System Side:** Purge

**P&ID Reference:** CKV 301

**Vendor:** Check-All

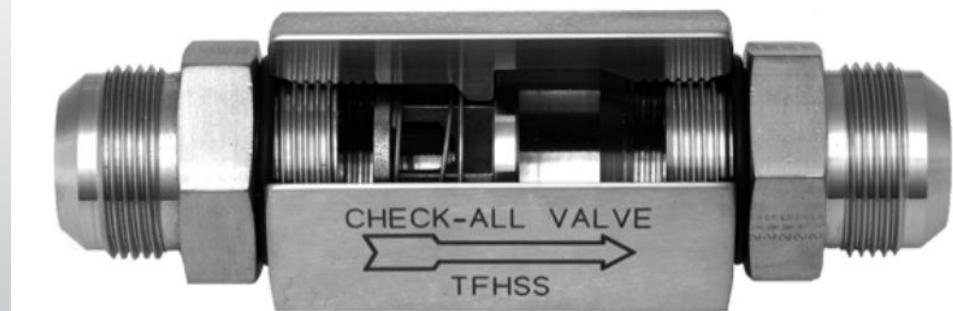
**Part Number:** TFDSSMT1.50SS

**Cv:** 3.1

**Inlet/Outlet:**  $\frac{1}{2}$  in. Male AN

**Max Pressure:** 3000 psig

**Price:** \$183.35



# Check Valve Selection



**Feed System Side:** LOx

**P&ID Reference:** CKV 101

**Vendor:** Check-All

**Part Number:** U3HSSTF1.50SS

**Cv:** 14.6

**Inlet/Outlet:** 1 in. Female NPT

**Max Pressure:** 3000 psig

**Price:** \$213.15

**Feed System Side:** Pressurant, LOx

**P&ID Reference:** CKV 002, CVK 102

**Vendor:** Check-All

**Part Number:** U3DSSTF1.50SS

**Cv:** 4.3

**Inlet/Outlet:**  $\frac{1}{2}$  in. Female NPT

**Max Pressure:** 3000 psig

**Price:** \$118.5

\*Reusing one check valve from Balerion (CVK 201)



# Solenoid Valve Selection



**Feed System Side:** Purge

**P&ID Reference:** SV 303, SV 304

**Vendor:** GC Valves

**Part Number:** H401GF24Z1DF5E

**Inlet/Outlet:**  $\frac{1}{2}$  in. Female NPT

**Max Pressure:** 2200 psig

**CV:** 1.5

**Price:** \$132.80

**\*Reusing SV-301 and SV-302 from Hydra**



# Needle Valve Selection



**Feed System Side:** Purge

**P&ID Reference:** NV 301

**Vendor:** McMaster

**Part Number:** 4559K34

**Inlet/Outlet:**  $\frac{1}{2}$  in. Female NPT

**Max CV:** 0.76

**Max Pressure:** 6000 psig

**Price:** \$57.95

**\*Reusing NV-201 and NV-302 from Hydra**



# Filter Selection



**Feed System Side:** Pressurant/Purge

**P&ID Reference:** FLTR-007,301

**Vendor:** Swagelok

**Part Number:** SS-8TF-60

**Inlet/Outlet:**  $\frac{1}{2}$  in. Swagelok Tube Fitting

**Filter Size:** 60 Micron

**Max Pressure:** 6000 psig

**Price:** \$182.27



**Feed System Side:** LOx

**P&ID Reference:** FLTR-101

**Vendor:** Chase Filters

**Part Number:** 23T-8P8P-150ST

**Inlet/Outlet:**  $\frac{1}{2}$  in. Female NPT

**Filter Size:** 60 Micron

**Max Pressure:** 6000 psig

**Price:** \$687.00



# Flow Meter Selection



**Feed System Side:** LOx, Fuel

**P&ID Reference:** FM-101,201

**Vendor:** Turbine Inc.

**Part Number:** HA0100-AN-NI

**Inlet/Outlet:** 1 in. AN

**Transmitter:** 4-20 mA

**Transmitter Part Number:** FIC712-2

**Flow Rate:** 5-60 GPM +/- 0.5% of flow rate

**Max Pressure:** 2000 psig

**Price:** \$1438



# Propellant Tank Selection



**Feed System Side/Fluid:** Pressurant

**Vendor:** Tri-Med

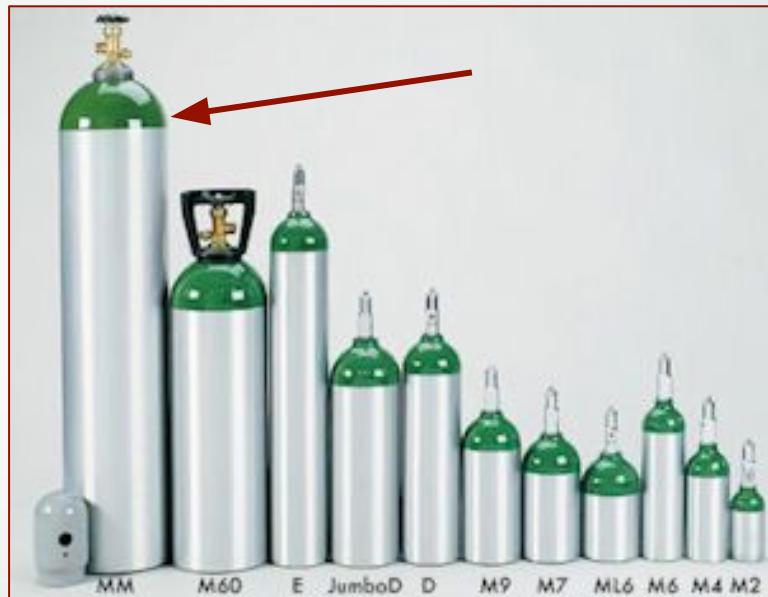
**Part Number:** CGA540 - SKU 20010

**Max Pressure:** 2200 psig

**Inlet/Outlet:**  $\frac{3}{8}$  in NPT & 1-1/8"-12 UNF

**Price:** \$399.99

**\*Will be tapping the tank + tank will be heat treated**



Common Oxygen Cylinder / Tank Size Chart:

	LITERS	PSI	HT	DIAM	LBS.
<b>M2</b>	40	2200	9.0	2.5	.74
<b>M4 = A</b>	113	2200	12.0	3.2	1.6
<b>M6 = B</b>	165	2200	15.0	3.2	2.8
<b>ML6</b>	165	2200	10.68	4.38	3.4
<b>M9 = C</b>	255	2000	14.1	4.38	3.7
<b>D</b>	425	2000	20.0	4.38	5.3
<b>Jumbo D</b>	640	2000	20.0	5.25	8.1
<b>E</b>	680	2000	29.0	4.38	7.9
<b>M60</b>	1738	2200	23.0	7.25	21.7
<b>MM</b>	3455	2200	35.75	8.0	38.6
<b>H (K)</b>	7842	2000	55.0	9.0	120

# Fuel Supply Tank Selection

**Feed System Side:** Fuel

**Vendor:** Scepter

**Part Number:** 08669

**Capacity:** 12 Gallons

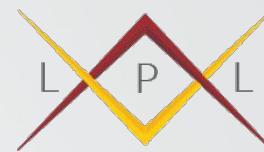
**Inlet/Outlet:**  $\frac{1}{4}$  in. NPT Pick-Up

**Material:** Polyethylene

**Price:** \$95.00



# Fuel Pump Selection



**Feed System Side:** Fuel

**P&ID Reference:** FP-001

**Vendor:** Grainger

**Part Number:** 38NH83

**Inlet/Outlet:**  $\frac{1}{2}$  in. NPT

**GPM:** 10 GPM

**Rated Fluid:** Liquid Fuels (i.e Kerosene, gasoline, diesel)

**Discharge Head:** 36 ft.

**Price:** \$182.04



# High Pressure Ball Valve Selection



**Feed System Side:** Pressurant

## ***Hand Ball Valve***

**P&ID Reference:** HBV-001

**Part Number:** CHPFNGSFM

**Inlet/Outlet:**  $\frac{1}{2}$  in. Female NPT

**Vendor:** Assured Automation

**Cv:** 28

**Max Pressure:** 3000 psig

**Price:** \$245.00

## ***Pneumatic Ball Valves***

**P&ID Reference:** PBV-001,002

**Part Number:** CRS329HP- $\frac{1}{2}$

**Inlet/Outlet:**  $\frac{1}{2}$  in. Female NPT

**Vendor:** CR-TEC

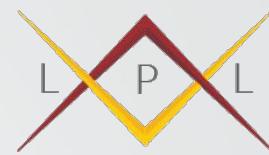
**Cv:** 19

**Max Pressure:** 3000 psig

**Price:** \$425.00



# Ball Valve Selection



**Feed System Side:** Pressurant/Fuel

**Inlet/Outlet:**  $\frac{1}{2}$  in. Female NPT

**Vendor:** Assured Automation

**Cv:** 28

**Max Pressure:** 2000 psig

## ***Hand Ball Valves***

**P&ID Reference:** HBV-002,003,004,201

**Part Number:** C26NRXM

**Price:** \$120.00

**\*Reusing 1 Handball Valve from Hydra (HBV-002)**

## ***Pneumatic Ball Valves***

**P&ID Reference:** PBV-003,004,005

**Part Number:** C26NRXC8SCC3F/C26NRXC8SOC3F

**Pneumatic Actuator:** C Series Dual Rack-n-Pinion

**Supply Air Pressure:** 80 Psig

**Price:** \$454.00

**\*Reusing 2 Pneumatic Ball Valves from Hydra (PBV-003,004)**

**Inlet/Outlet:** 1 in. Female NPT

**Vendor:** Assured Automation

**Cv:** 87

**Max Pressure:** 2000 psi

## ***Pneumatic Ball Valves***

**P&ID Reference:** PBV-201

**Part Number:** E26NRXC8SCC3F

**Pneumatic Actuator:** C Series Dual Rack-n-Pinion

**Supply Air Pressure:** 80 Psig

**Max Pressure:** 2000 psig

**Price:** \$598.00

**\*Reusing Pneumatic Ball Valve from Hydra**



# Cryogenic Ball Valve Selection



**Feed System Side:** Pressurant/Ox

**Inlet/Outlet:**  $\frac{1}{2}$  in. Female NPT

**Vendor:** Assured Automation

**Cv:** 18

**Max Pressure:** 3000 psig

## *Hand Ball Valves*

**P&ID Reference:** HBV-005,101

**Part Number:** HPF51SS-1-FGF-L-215

**Price:** \$609.20

## *Pneumatic Ball Valves*

**P&ID Reference:** PBV-006,102

**Part Number:** HPF51SS-1-FGF-G-15

**Pneumatic Actuator:** C Series Dual Rack-n-Pinion

**Supply Air Pressure:** 80 Psig

**Price:** \$1032.00

**Inlet/Outlet:** 1 in. Female NPT

**Vendor:** Assured Automation

**Cv:** 72

**Max Pressure:** 3000 psig

## *Pneumatic Ball Valves*

**P&ID Reference:** PBV-101

**Part Number:** HPF51SS-1-FGF-G-25

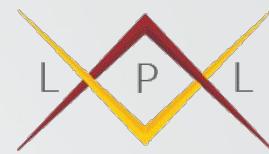
**Pneumatic Actuator:** C Series Dual Rack-n-Pinion

**Supply Air Pressure:** 80 Psig

**Price:** \$1231.00



# PRV Orifice Sizing



The following equations was used to size the orifice:

Set pressure: 1889.7 psia

Discharge Coefficient (K) = .75

Molecular Weight = 28 kg/kmol

Specific Heat Ratio = 1.4

T = 316.5 K (110 °F)

W = 4859 (lbs/h)

Nozzle Constant (C, function of  $\gamma$ ) = 356

$$A = \frac{W \sqrt{TZ}}{CK P_1 K_b \sqrt{M}}$$



This provided an orifice sizing of: **0.2352 inches.**

# PRV Selection



**P&ID Reference:** PRV-003,004

**Part Number:** PRVN4F-02-2-VV

**Inlet/Outlet:** ½ in. Female NPT

**Vendor:** Swagelok

**Cv:** 4.36

**Max Pressure:** 6000 psig

**Orifice Diameter:** 0.45 in

**Price:** \$300



# Rupture Disk Selection



**P&ID Reference:** RD-001,002

**Part Number:** NSP-2002-2

**Inlet/Outlet:**  $\frac{1}{4}$  in. Female NPT

**Vendor:** NLB Corp.

**Set Pressure:** 2000 psig

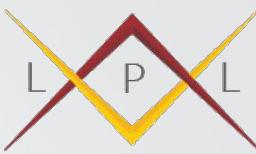
**Diameter:** 0.25 in

**Price:** \$50.20

→ Re-use existing Swagelok disk housing from Hydra Dev-1, swap-in new disk with correct set pressure



# Regulator Selection



**Feed System Side:** Pressurant

**P&ID Reference:** REG-001 & REG-002

**Vendor:** Swagelok

**Part Number:** RSHN4-02-VVK-GN2

**Cv:** 1.84

**Inlet/Outlet:**  $\frac{1}{2}$  in. Female NPT

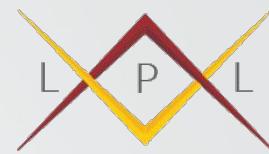
**Max Inlet Pressure:** 5800 psig

**Pressure Control Range:** 0 - 2175 psig

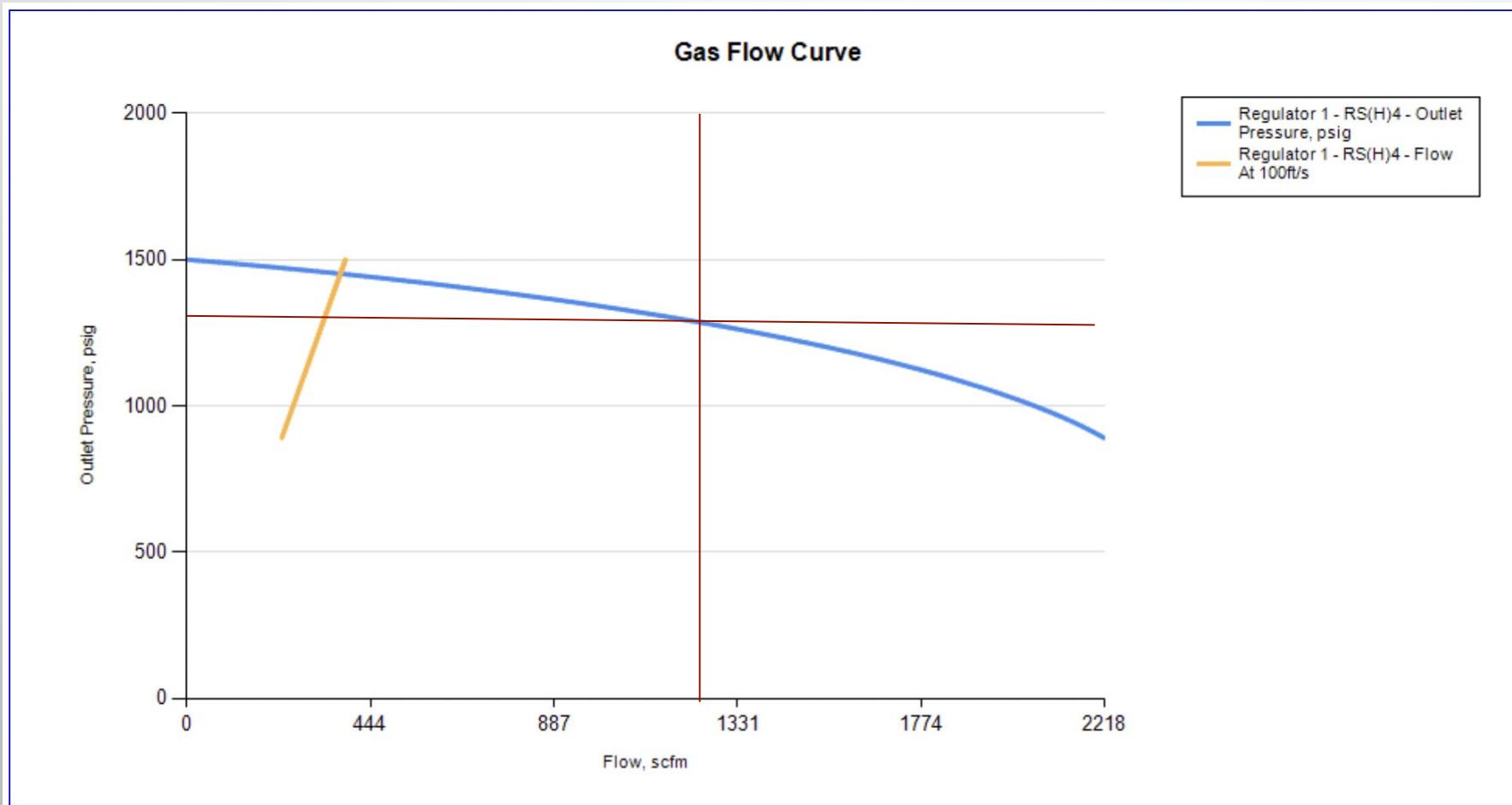
**Price:** \$2148.21



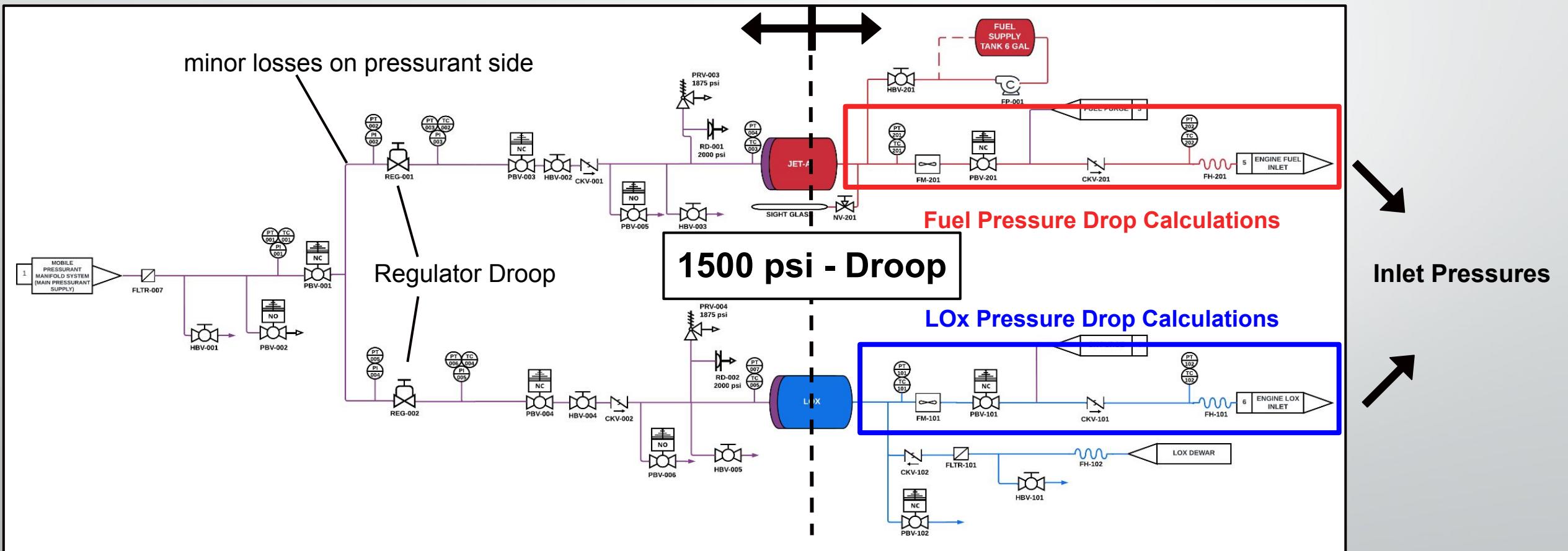
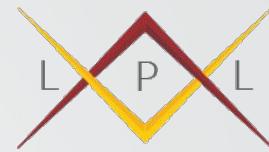
# Droop



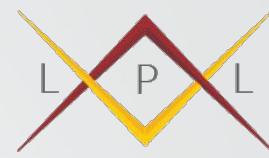
**Droop:** A decrease in outlet pressure caused by an increase in flow rate to a pressure reducing regulator.



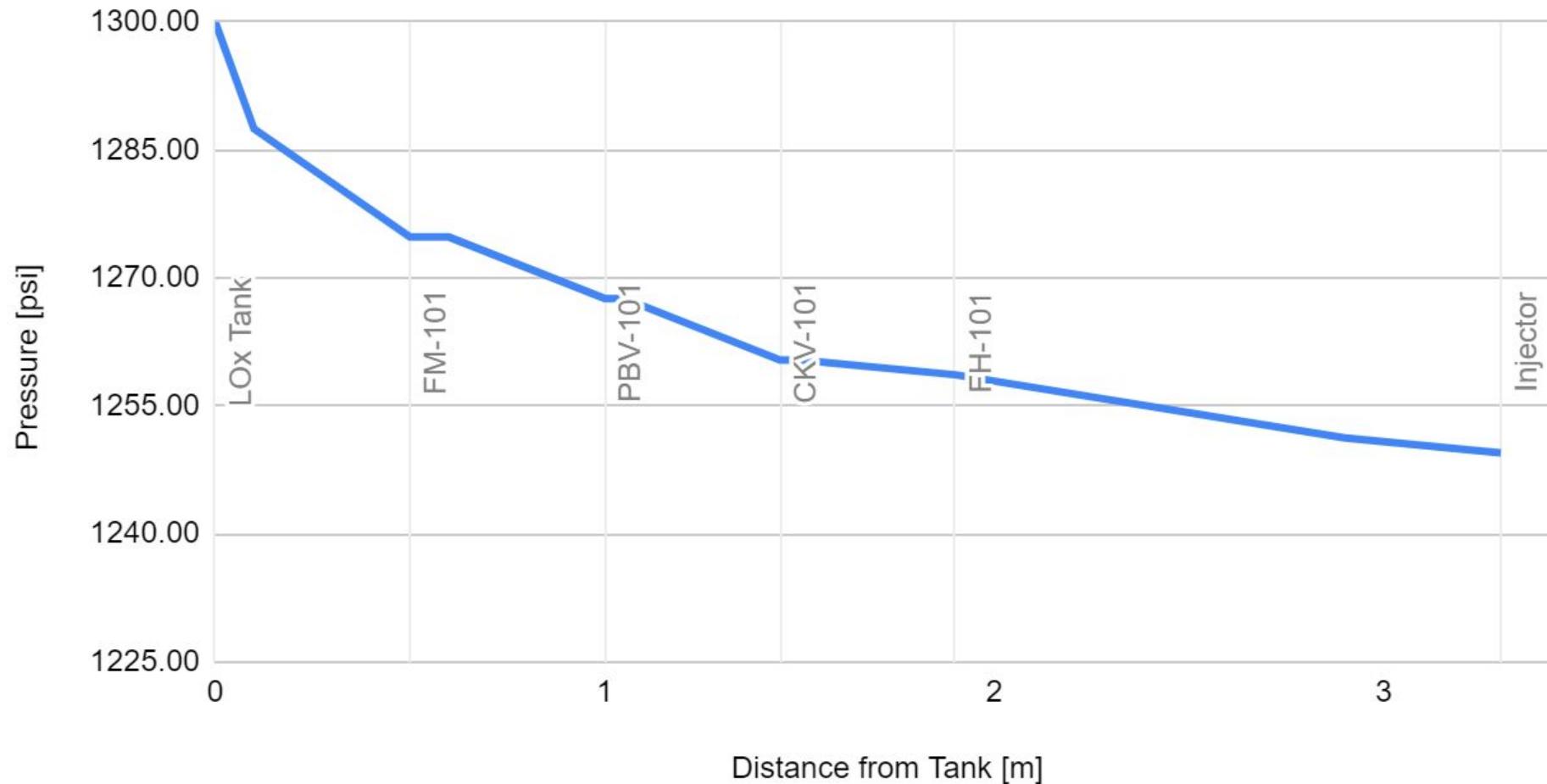
# Pressure Drop



# Pressure Drop - LOx



LOx Pressure Loss

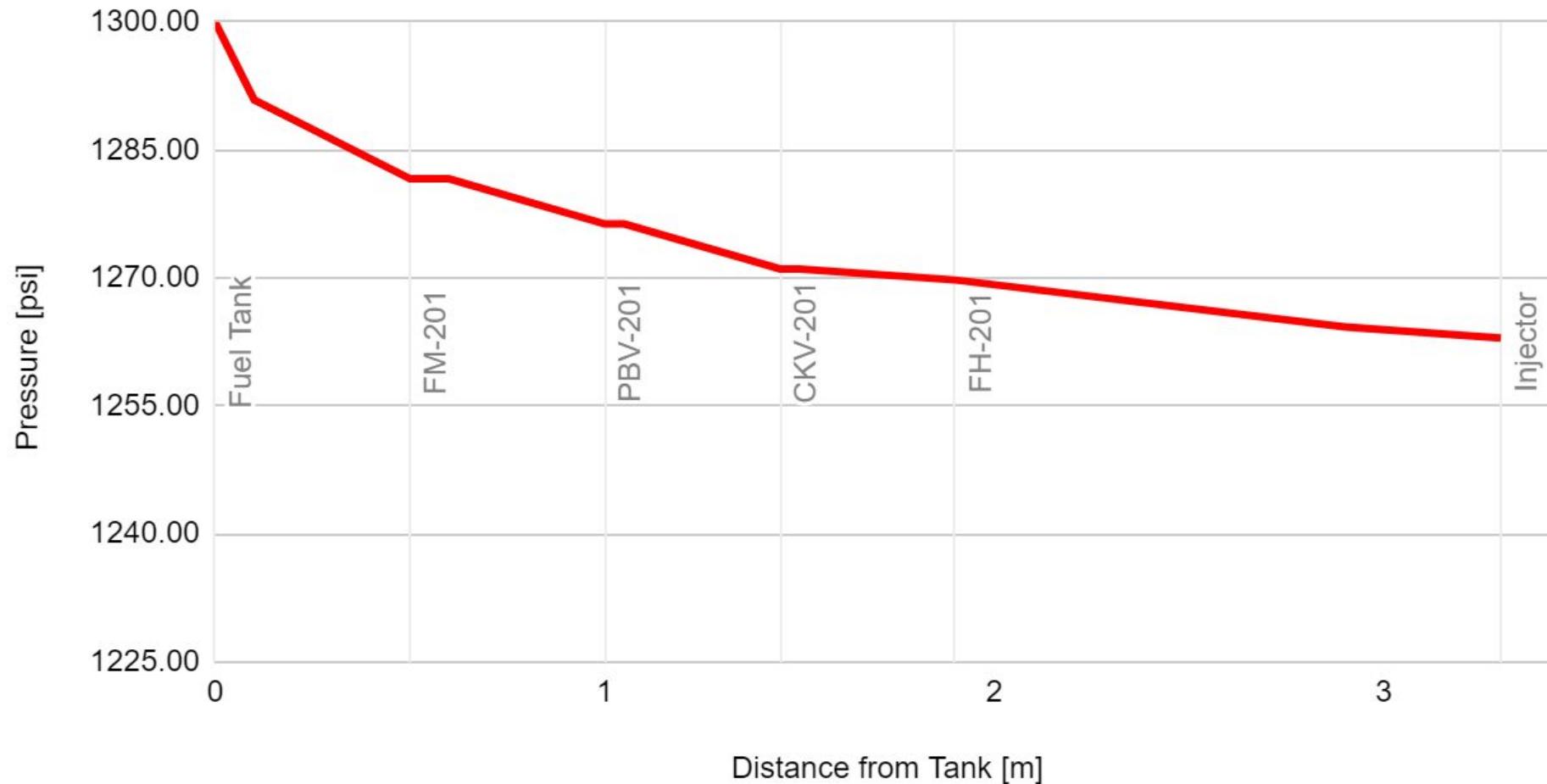


→ 1249 psi

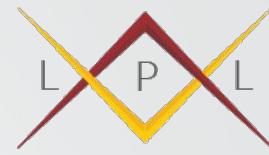
# Pressure Drop - Fuel



Fuel Pressure Loss



→ 1262 psi



## Pressure Transducers

- PT's on cryo lines will have a stand-off to avoid the need for cryo rated PT's

**P&ID Reference:** PRV 001, 002, 003, 004, 005, 006, 007, 101, 102, 201, 202, 401

**Part Number:** PX359

**Inlet/Outlet:**  $\frac{1}{4}$  in. Male NPT

**Vendor:** Omega

**Max Pressure:** 1000 psia

**Price:** \$371.01

## Thermocouples

- T-type TCs for all cryo wetted components and K-type for the rest

**P&ID Reference:** TC 001, 002, 003, 004, 005, 101, 102, 201, 202

**Part Number:** TQSS

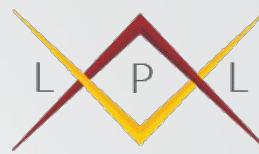
**Inlet/Outlet:**  $\frac{1}{4}$  in. Male NPT

**Vendor:** Omega

**Max Temperature:** 1150 Celsius

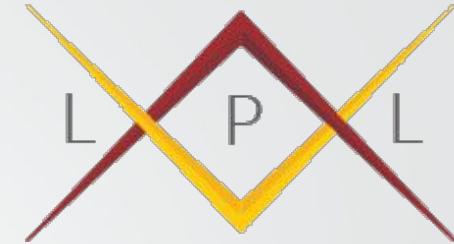
**Price:** \$50

# Feed System Cost



Component Cost Before Savings	\$28,787.41
Component Cost After Savings	<b>\$27,289.01</b>
Component Savings	\$1,498.40
Fitting Cost Before Savings	\$5,614.51
Fitting Cost After Savings	<b>\$5,232.88</b>
Fitting Savings	\$381.63
Total Cost	<b><u>\$32,521.89</u></b>

\*Potential Savings from PTs: \$4452.12



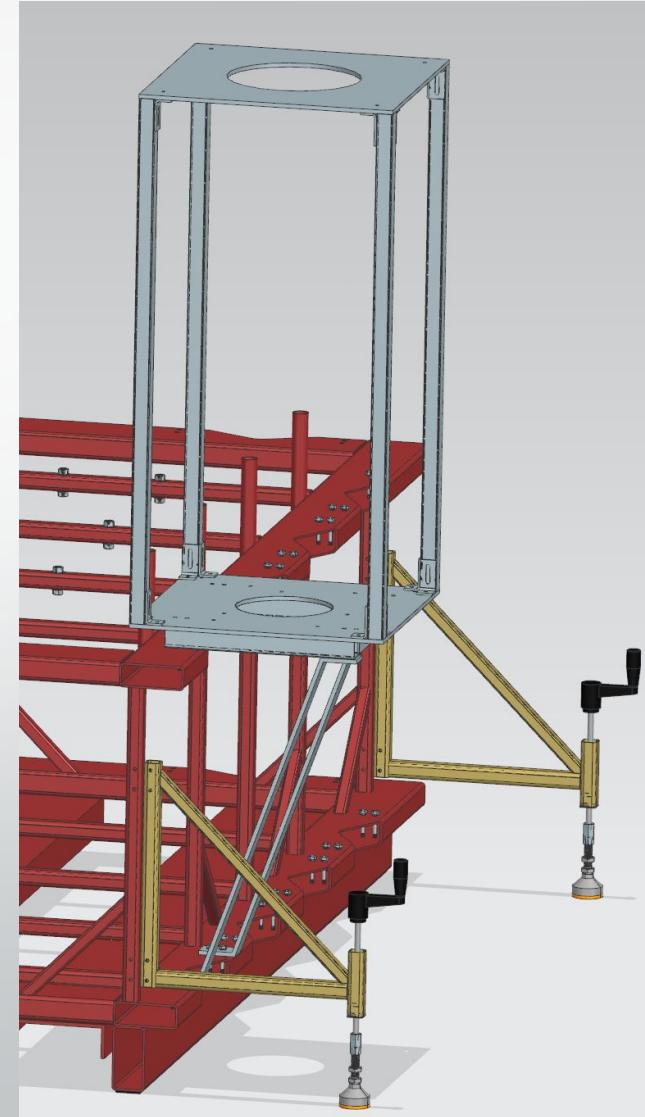
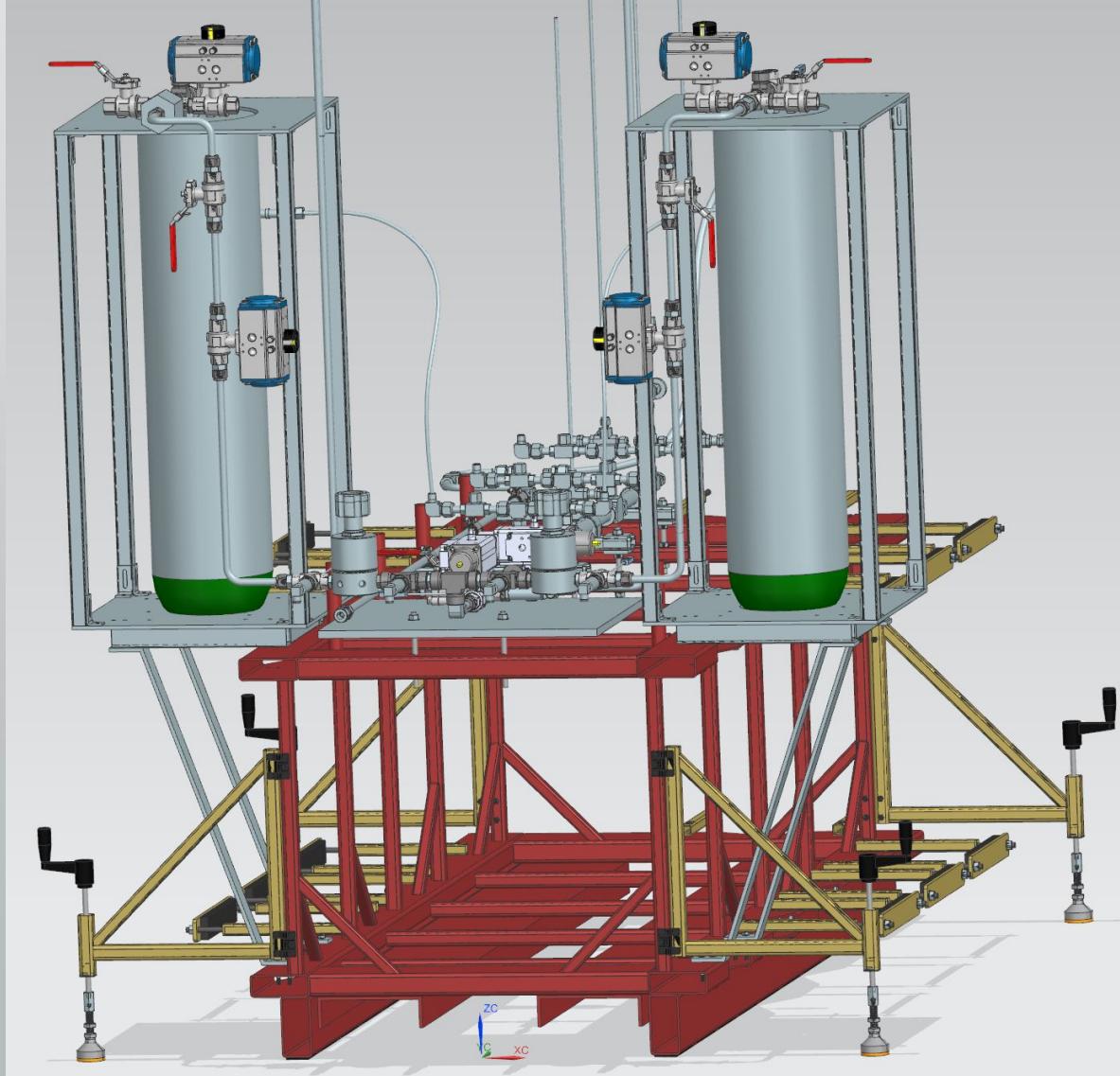
# Tank Support Structure

Liquid Propulsion Laboratory

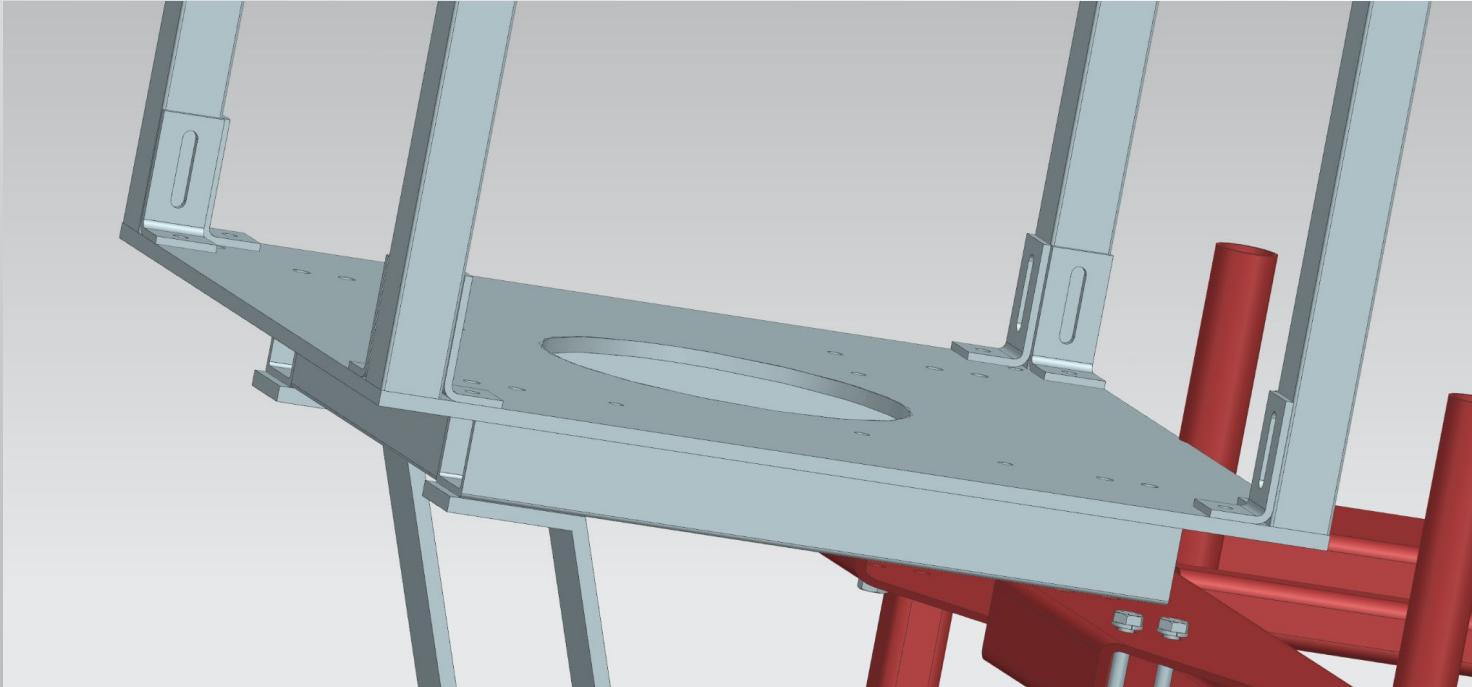


**USCViterbi**  
School of Engineering

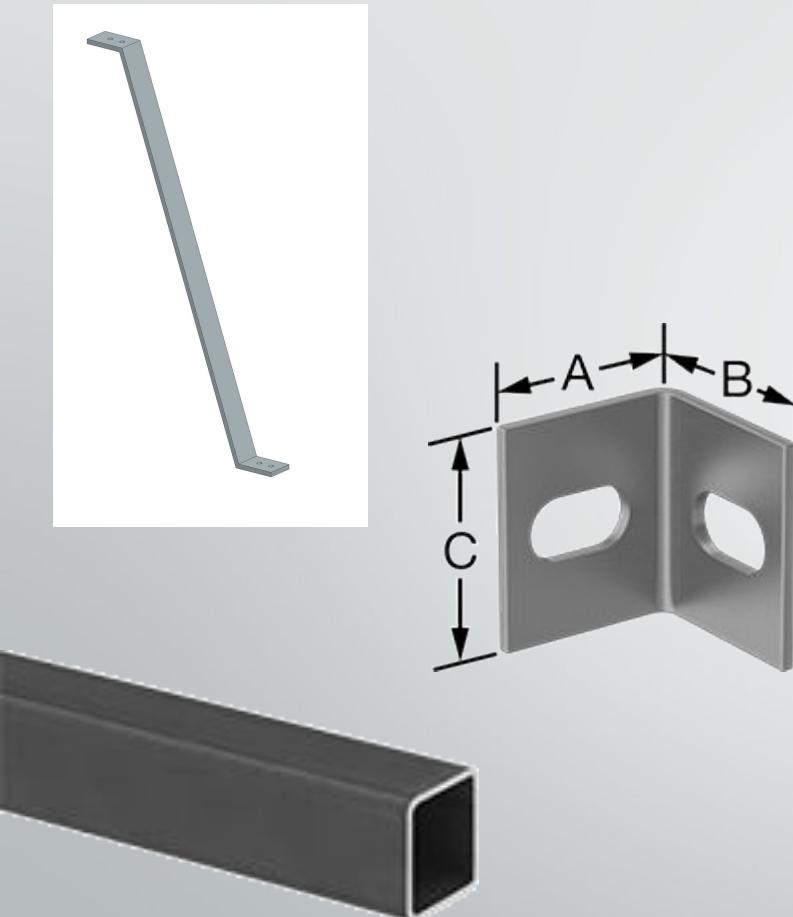
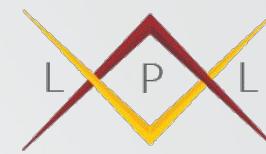
# Tank Structure



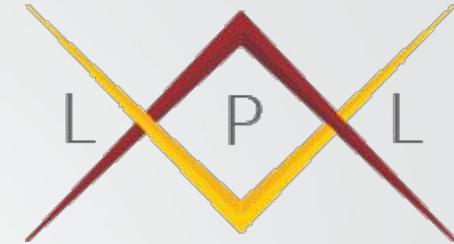
# Tank Structure



# Tank Structure



Component	Vendor	Amount	Price/Unit	Total Price
Square Tubing 3 ft (Vertical Column)	McMaster	4	\$7.69	\$30.76
Angle Brackets	McMaster	16	\$9.01	\$144.16
Steel Plates	McMaster	2	\$67.70	\$135.40
Square Tubing (Tank Platform)	McMaster	1	\$7.69	\$7.69
Steel Bar (Truss)	McMaster	2	\$33.72	\$67.44
Bolts	McMaster	48	\$13.86 (50 pc.)	\$13.86
Nut	McMaster	48	\$8.95 (100 pc.)	\$8.95
<b>Total (excluding machining costs)</b>				<b>\$408.26</b>



# Engine Mount

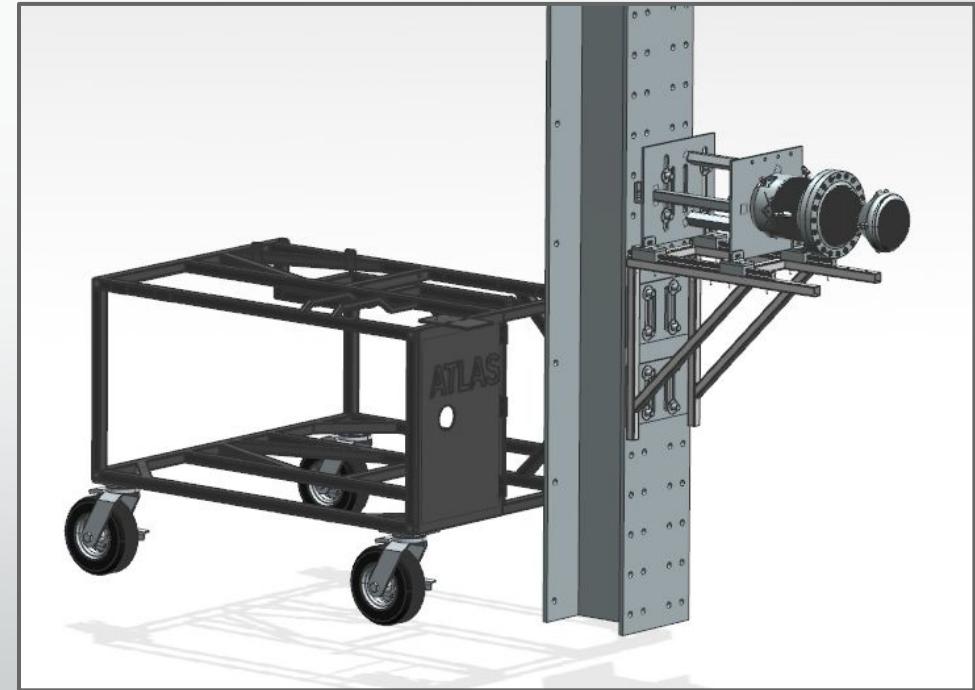
## Liquid Propulsion Laboratory



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School of Engineering

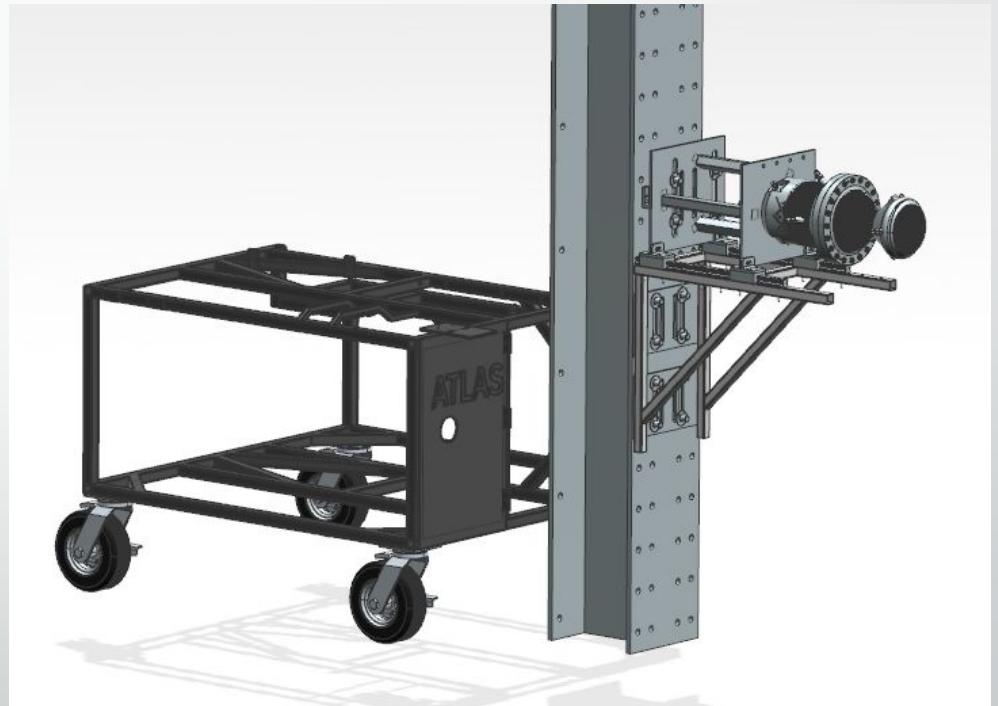


# New Horizontal Engine Mount



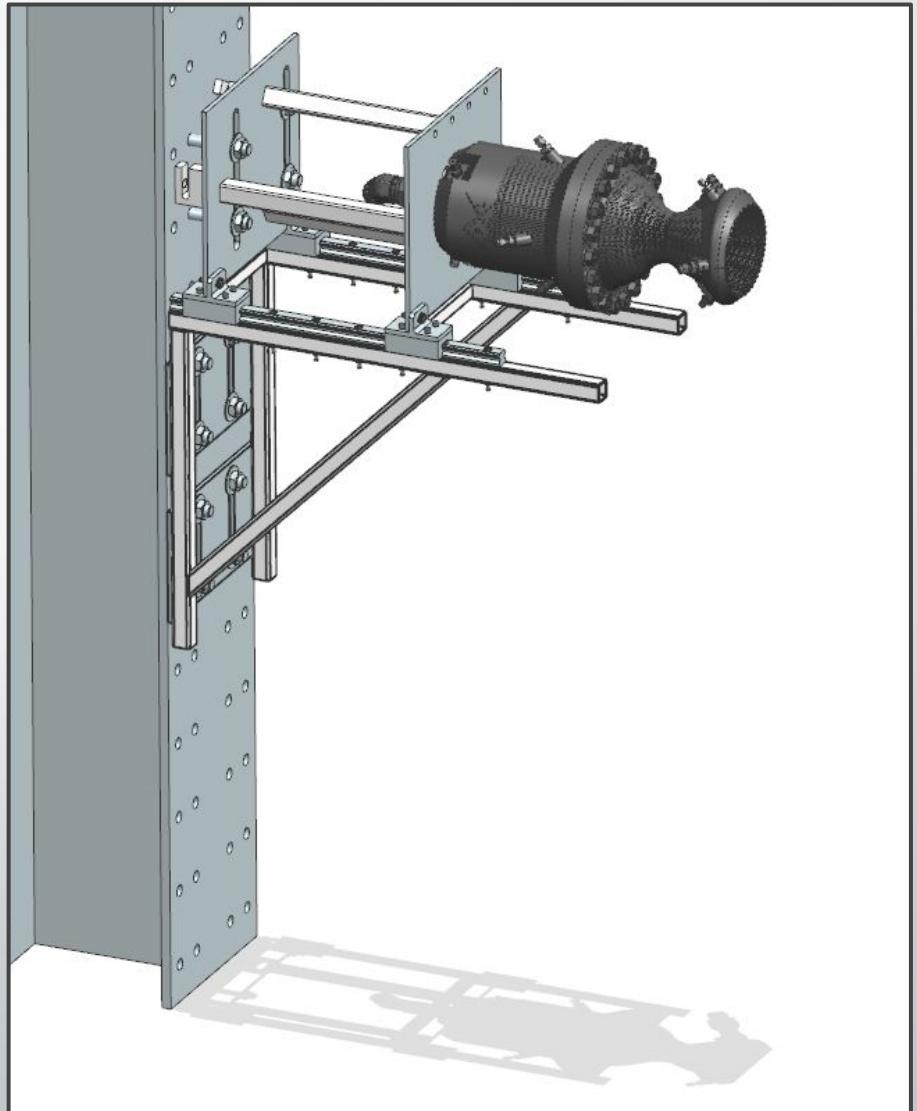
# Motivation

- Reduces complexity of feed system
  - Previously needed to bring both main lines, both purge lines, pneumatic lines, and DAQ wiring 6-8 ft up to Balerion
- Eliminates need for flame diverter reduces size of blast shield
- Why did we have vertical in the first place?
  - Based on Balerion history wanted to minimize chance of a hard start
    - No longer pre-chilling

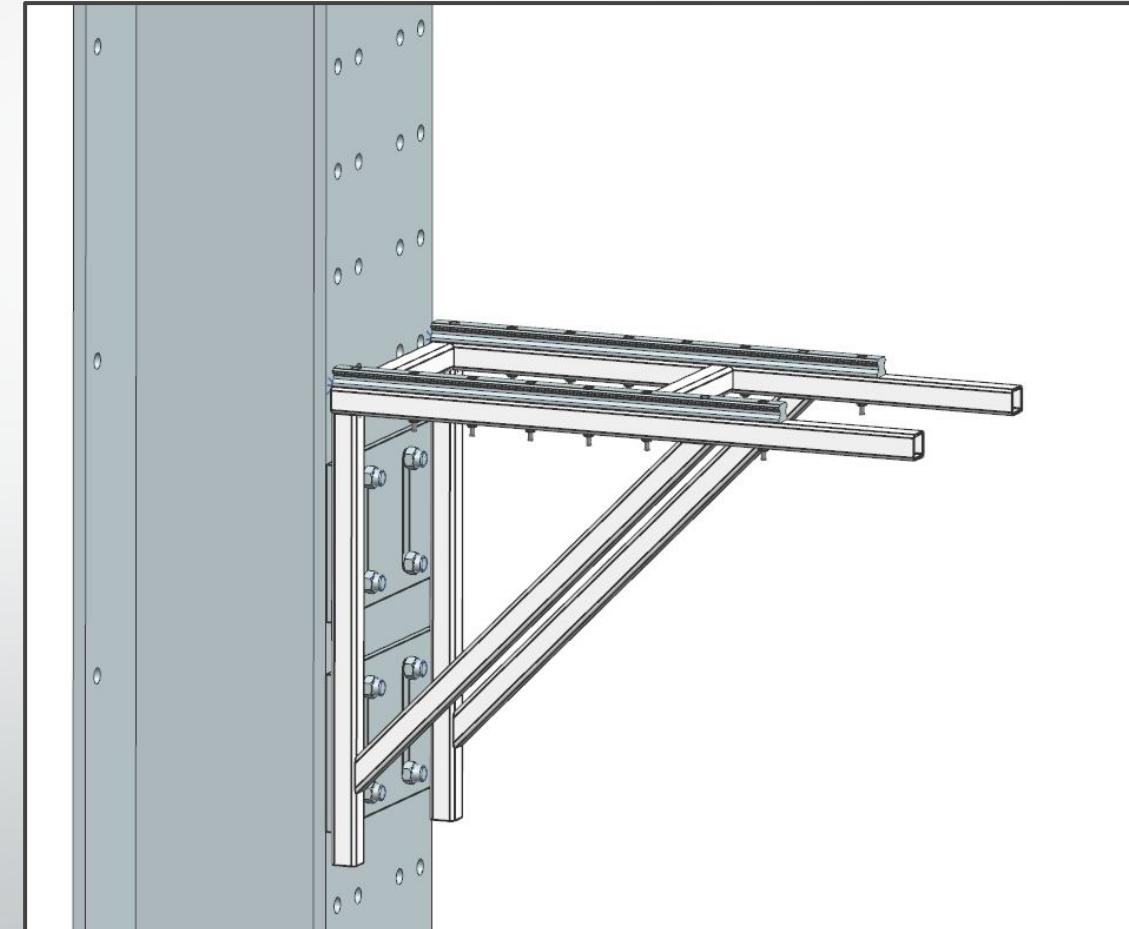
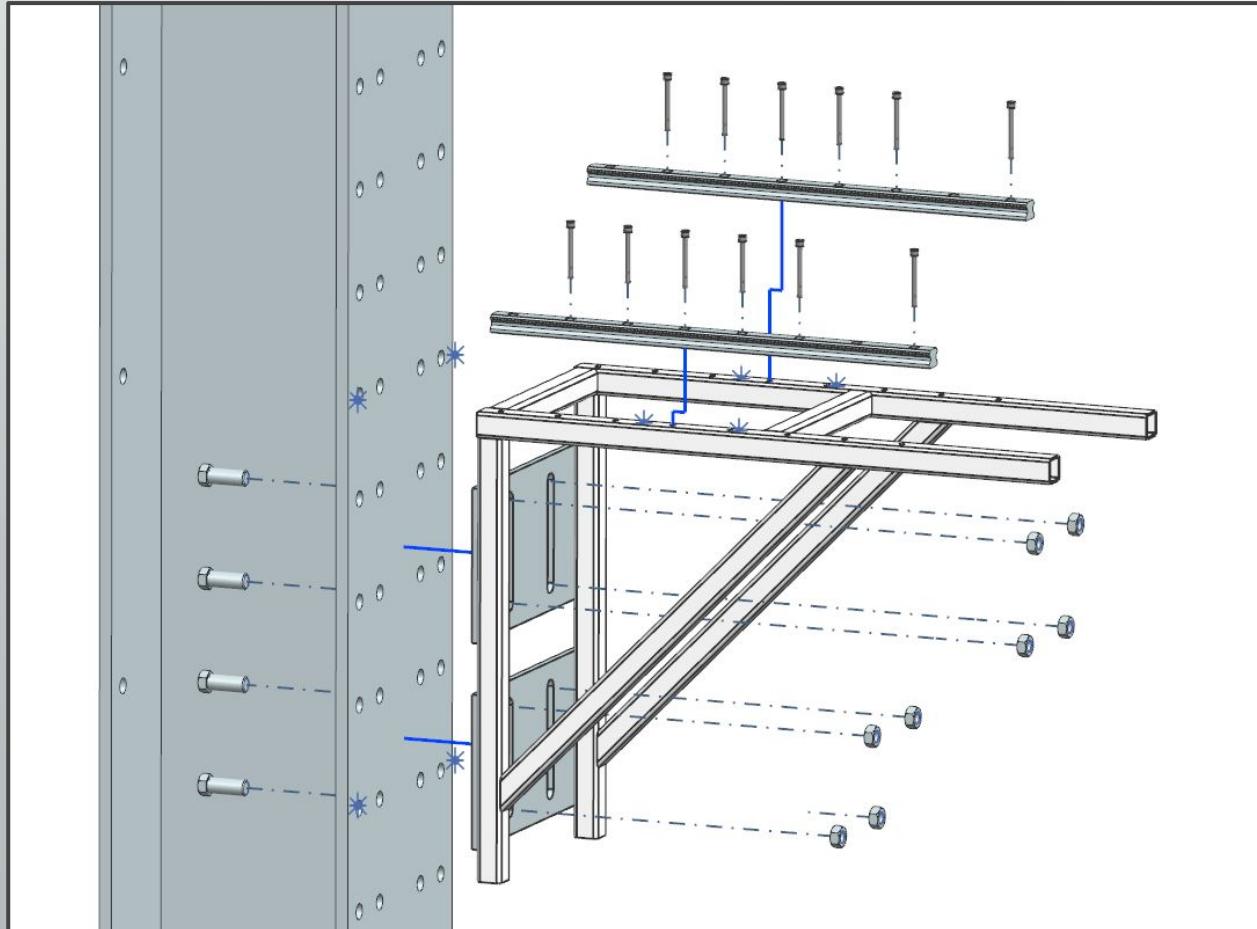


# Requirements

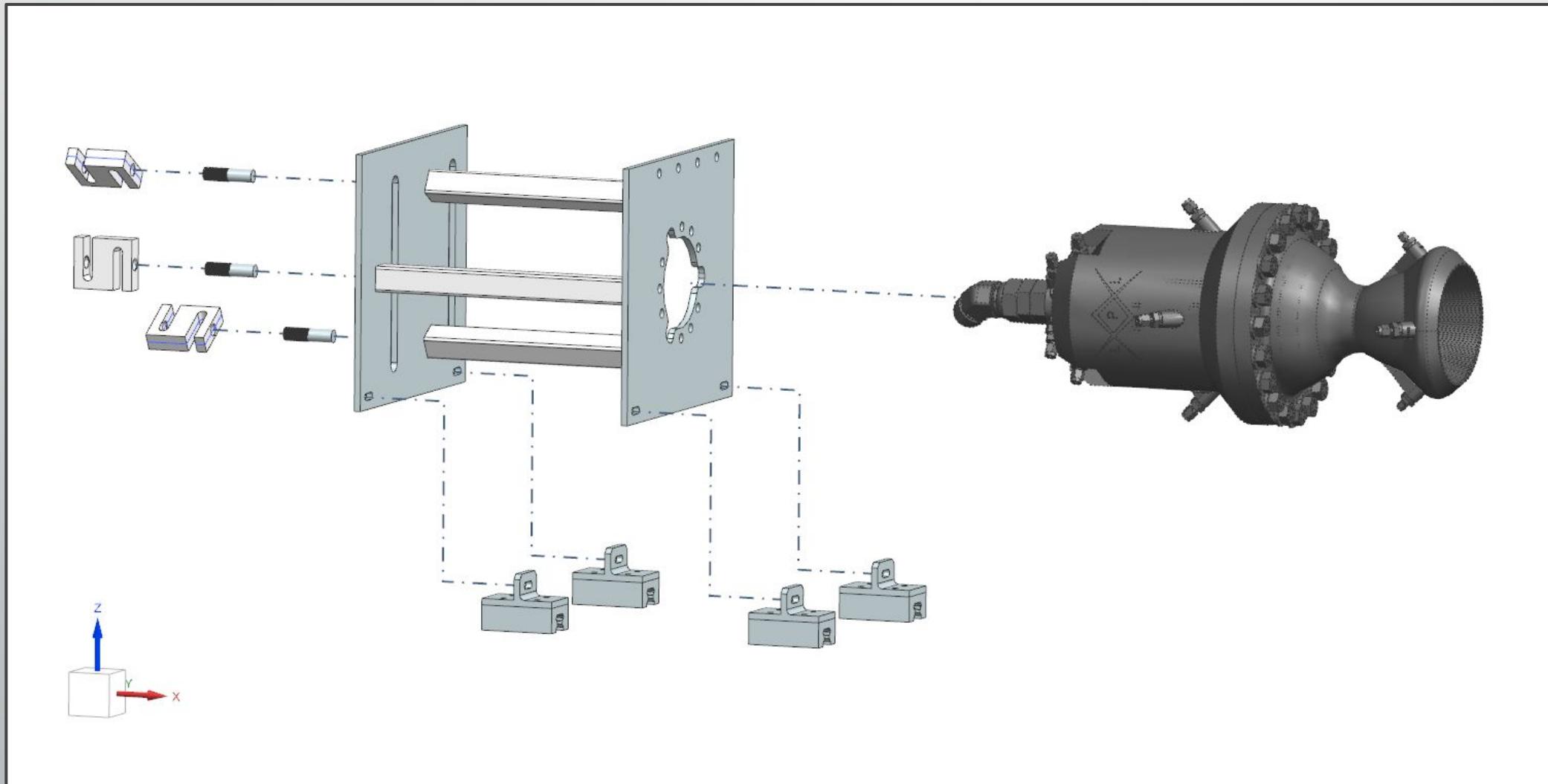
- Engine Mount shall...
  - Interface with FAR's Extra Large I-Beam
  - Interface with Balerion engine
  - Create even preload for load cells
  - Be liftable by 1-2 People
  - Withstand Balerion's Max Thrust (with 2.25 FOS)



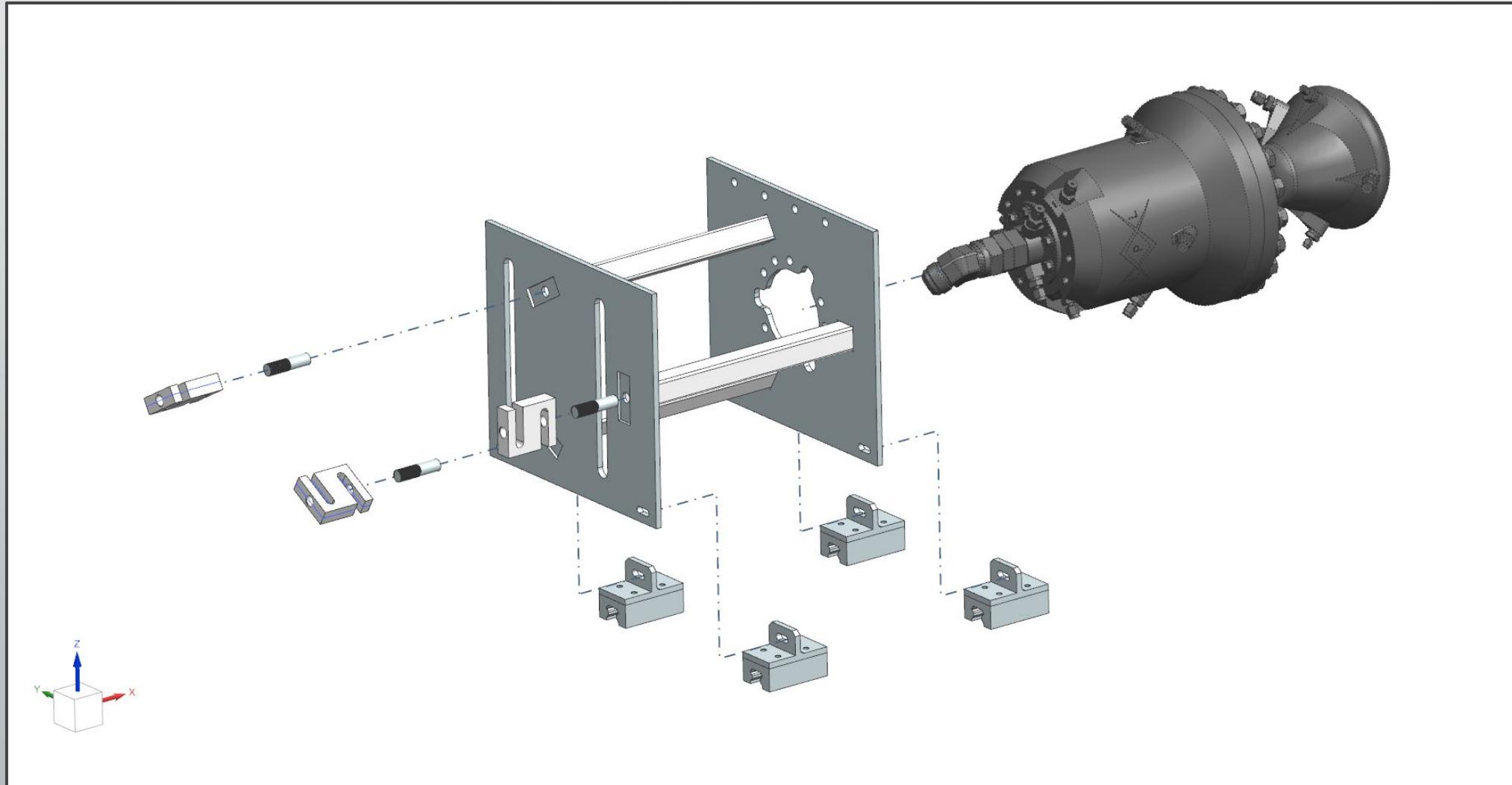
# Design Overview - Rail Platform



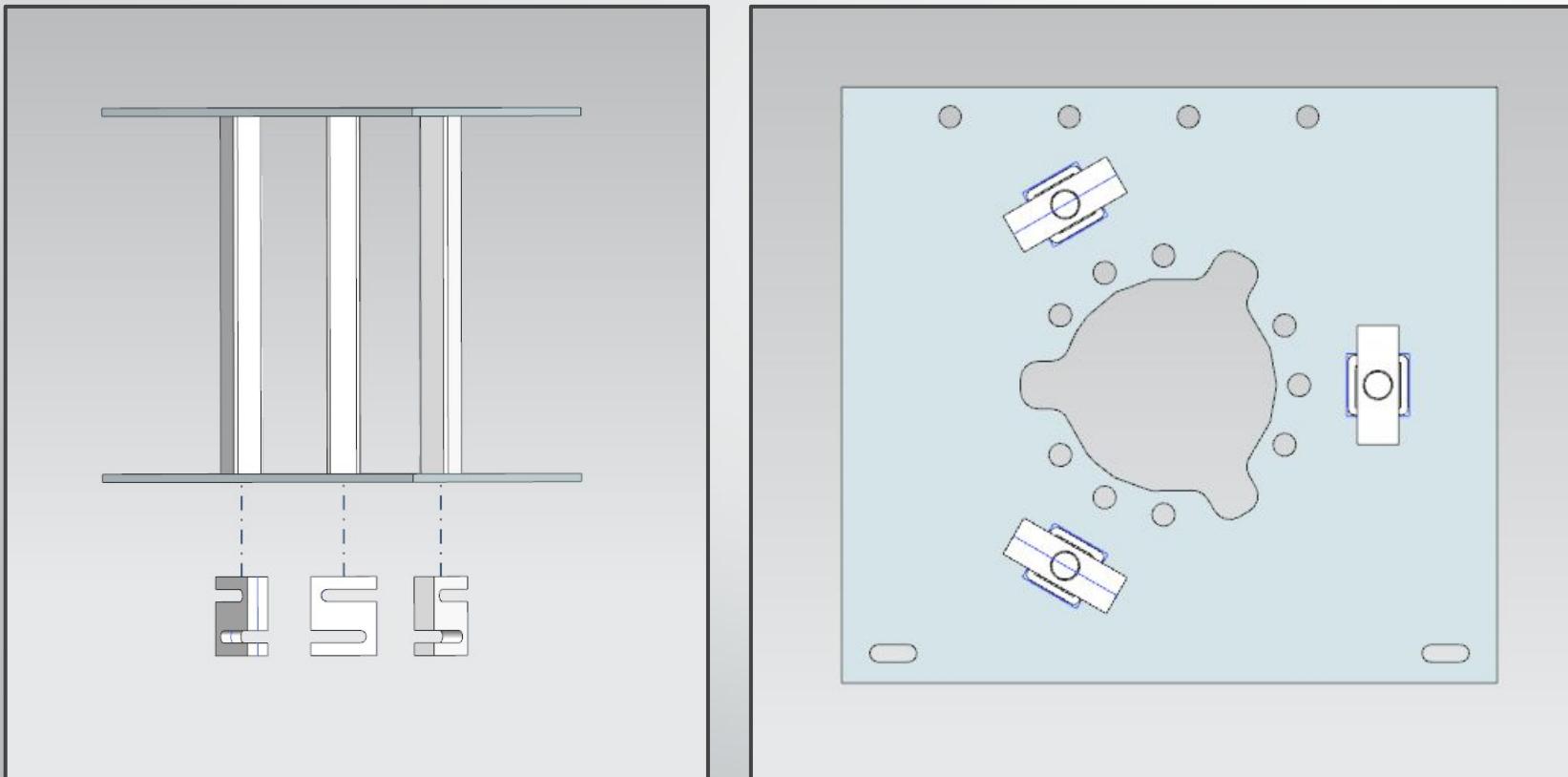
# Design Overview - Milk Stool



# Design Overview - Milk Stool



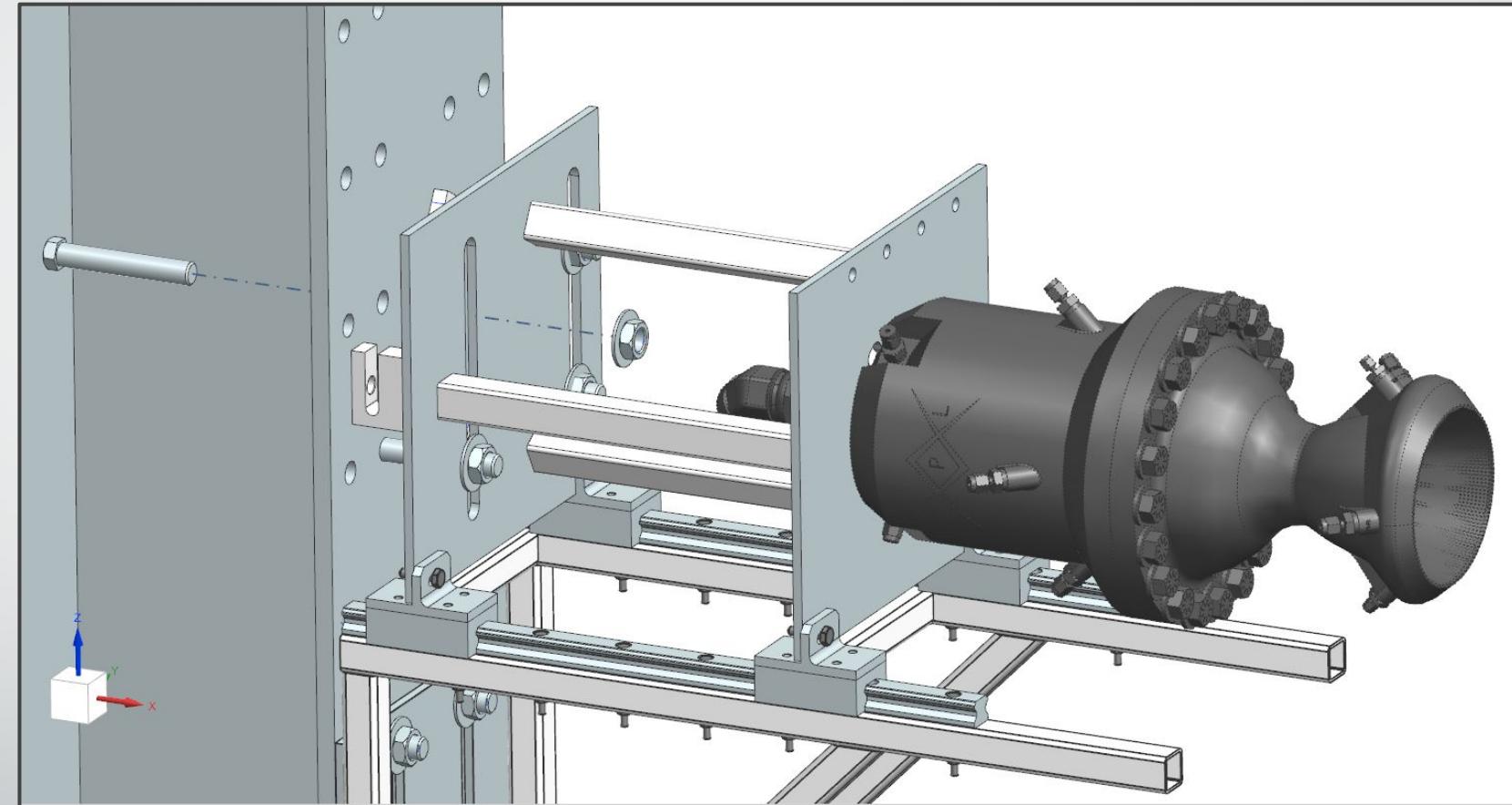
# Design Overview - Load Cells



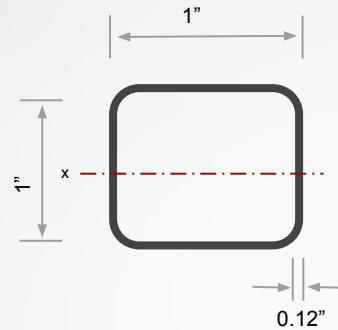
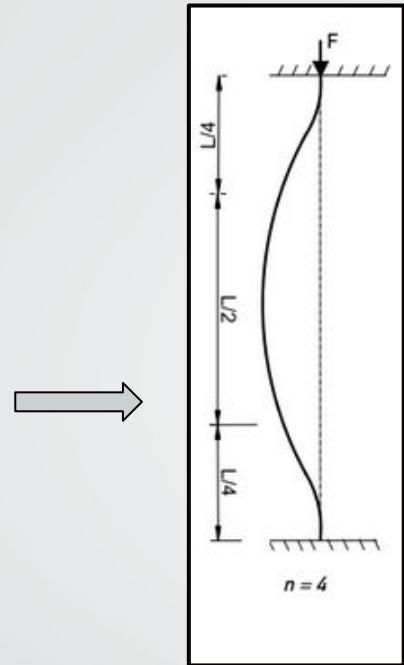
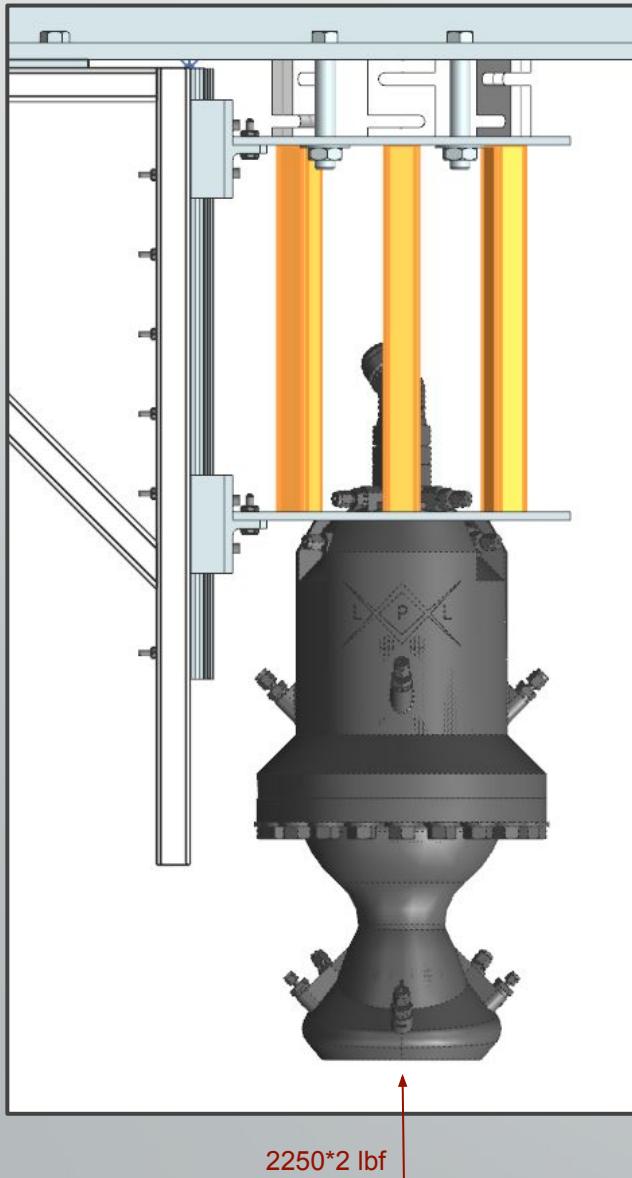
- Redundancy
- Off-axis loading measurement
- Parallel arrangement reduces cost as cells experience  $\frac{1}{3}$  of thrust, lowering load rating

# Design Overview - Interface

- Ball bearing carriages allow for untampered measurement of thrust (mitigates friction)
- Slots allow for vertical misalignments
- Use of UNF bolts to prevent loosening during engine fire



# Structural Analysis - Hand Calcs



$$\begin{aligned}n &= 4 \\E_{A36} &= 29,000 \text{ ksi} \\L &= 11 \text{ in} \\F_m &= 1500 \text{ lbf}\end{aligned}$$

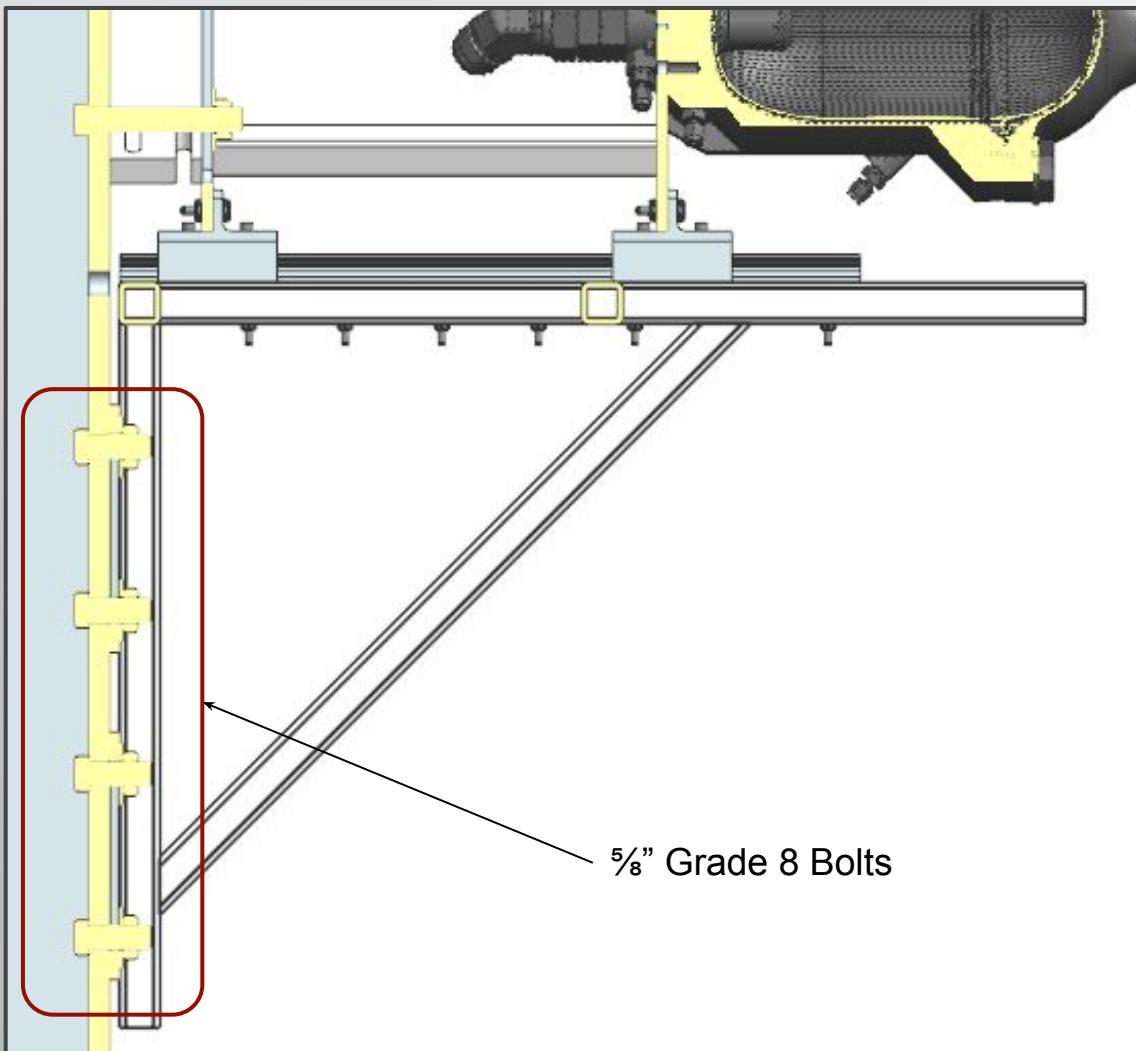
$$I_{xx} = \frac{bh^3}{12} = \frac{(1^4 - 0.76^4)}{12} = 0.055 \text{ in}^4$$

$$F_{\text{crit}} = n \pi^2 E_{A36} I_{xx} / L^2 = 1577 \text{ kip}$$

$$\boxed{\text{SF}_{\text{buckling},xx} = F_{\text{crit},xx} / F_m = 346}$$

Each column can support x346 its expected load before buckling occurs

# Structural Analysis - Hand Calcs



$$\sigma_{y, \text{grade 8 steel}} = 150 \text{ kip}$$

$$\text{Tensile Stress Area} = 0.226 \text{ in}^2$$

$$\mu_{\text{static, steel-steel}} \approx 0.7$$

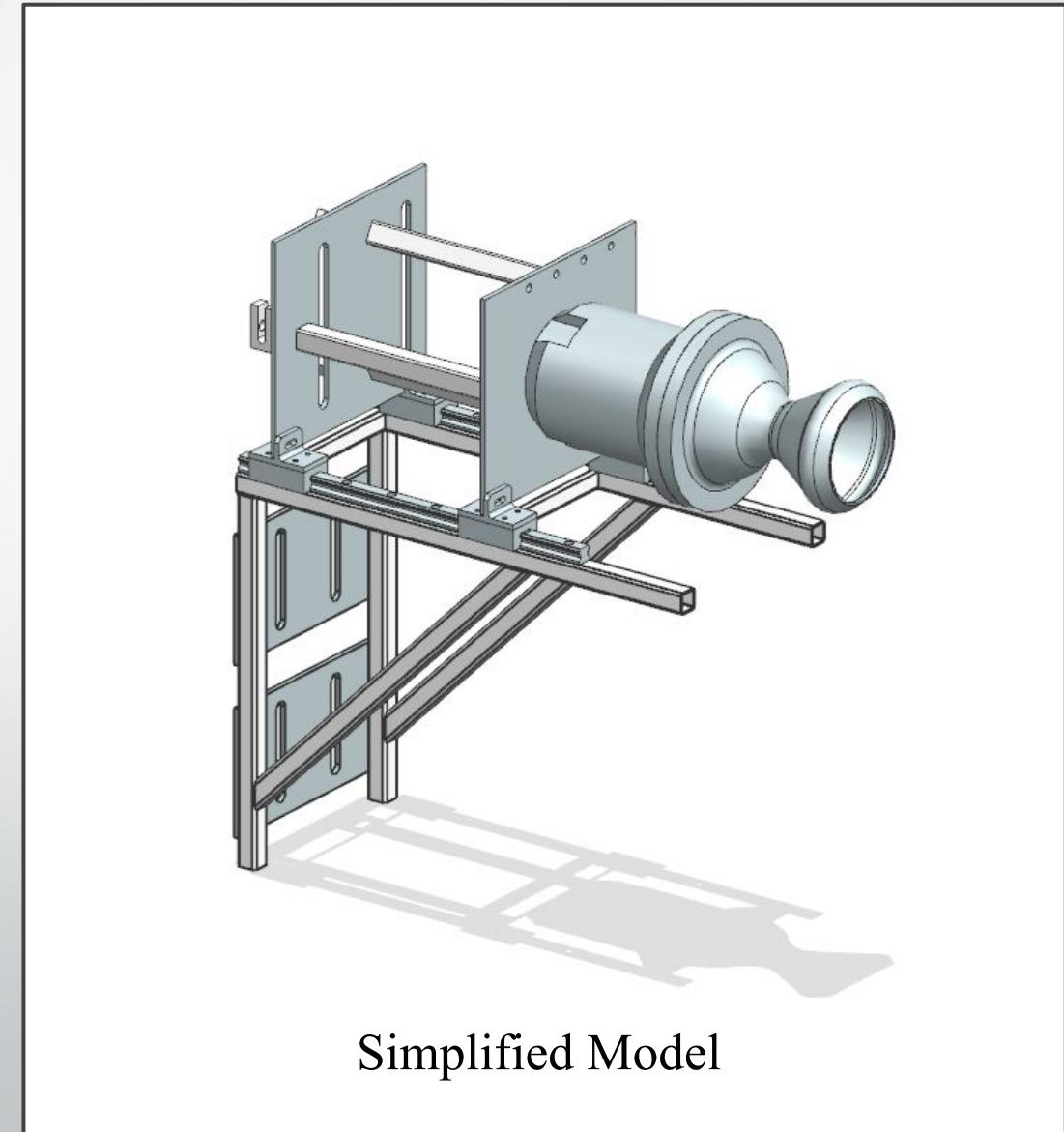
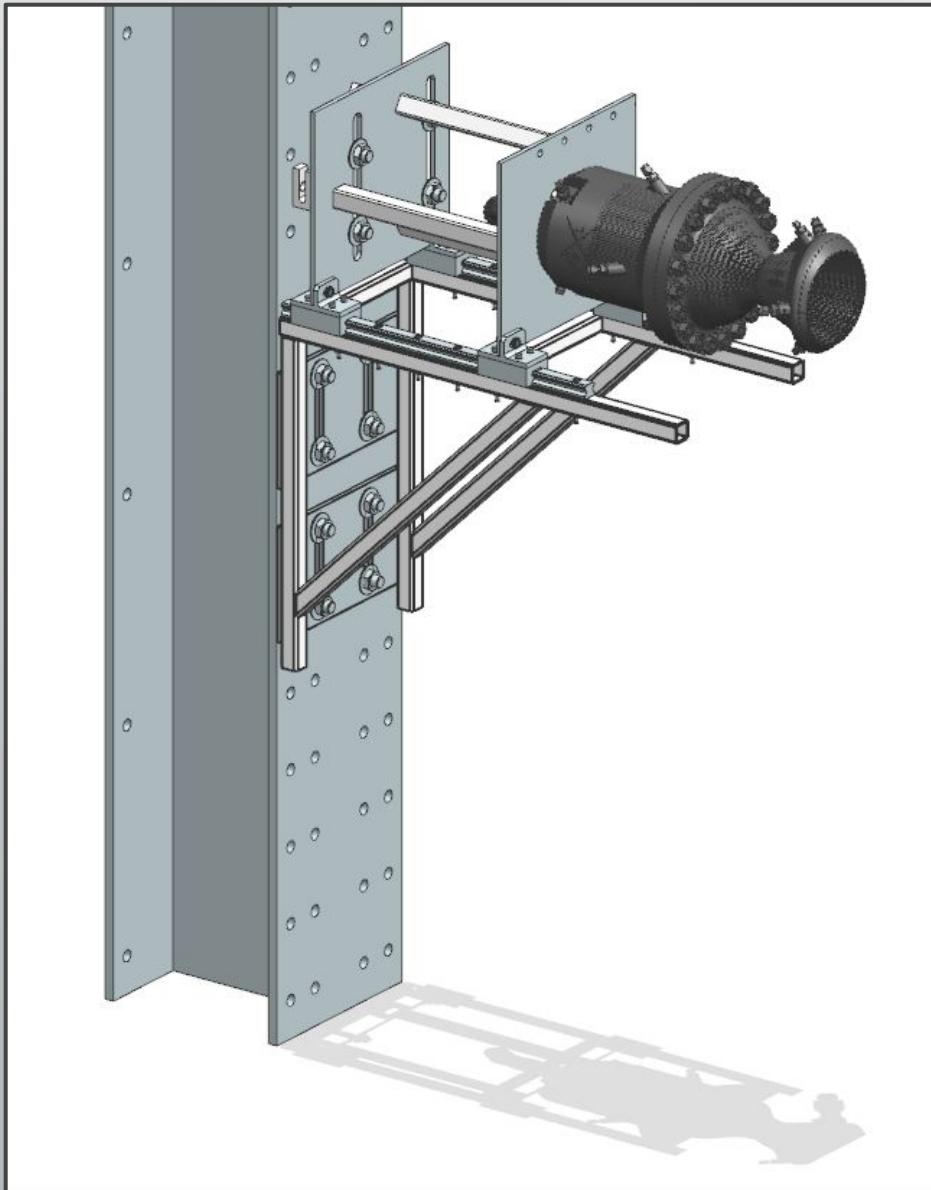
$$K = 0.2$$

$$\begin{aligned}\text{Preload Force} &= (75\%) * \sigma_{y, \text{grade 8 steel}} * (\text{Tensile Stress Area}) \\ &= (0.75) * (150 \text{ kip}) * (0.226 \text{ in}^2) \\ &= 25,430 \text{ lbf}\end{aligned}$$

$$\begin{aligned}\text{Installation Torque} &= K * (\frac{5}{8}'' \text{ nominal diameter}) * (\text{Preload Force}) \\ &= (0.2) * (0.625 \text{ in}) * (25,430 \text{ lbf}) \\ &= 2,543 \text{ in-lbf}\end{aligned}$$

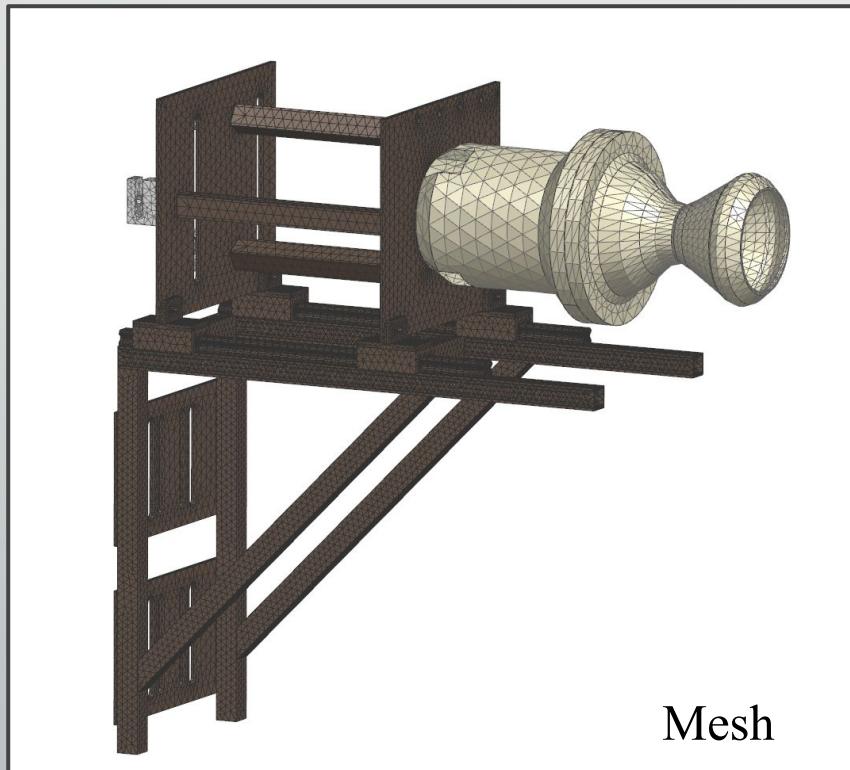
$$\begin{aligned}\text{Clamping Friction} &= (\# \text{ of bolts}) * (\text{Preload Force}) * \mu_{\text{static, steel-steel}} \\ &= (8) * (25,430) * (.7) \\ &= 142.4 \text{ kip}\end{aligned}$$

# Structural Analysis - FEA

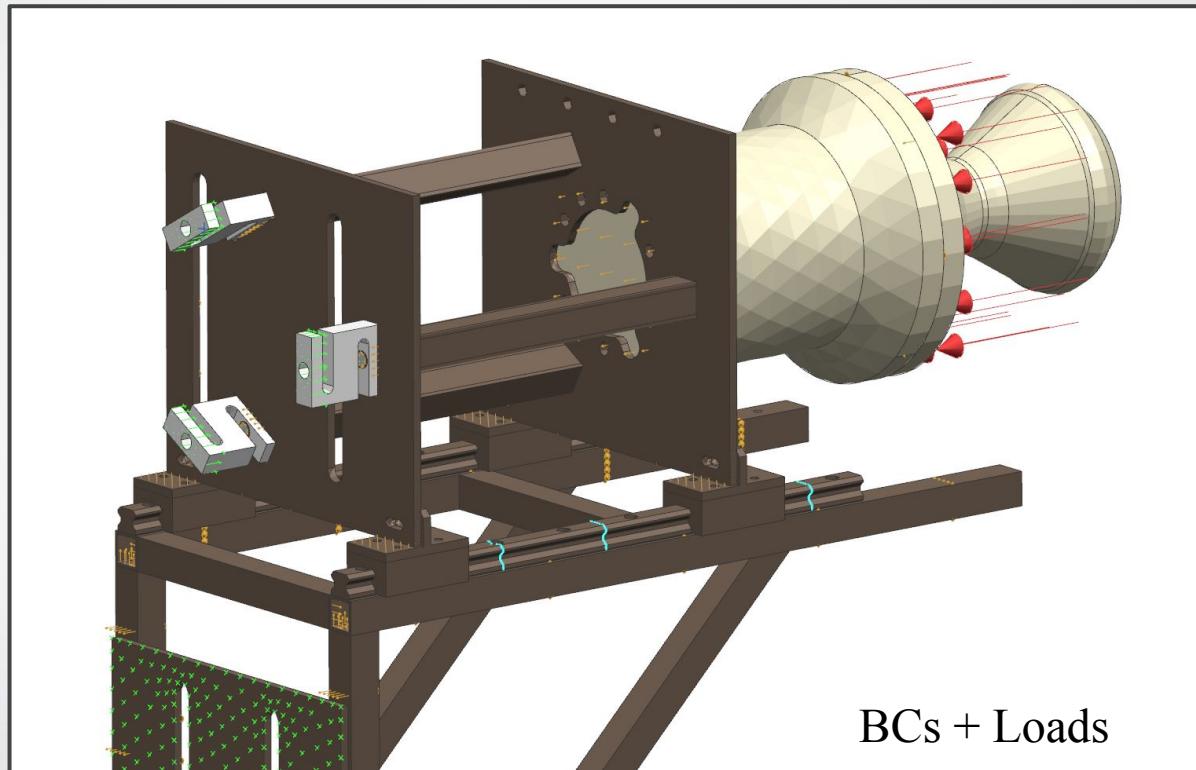


Simplified Model

# Structural Analysis - FEA



Mesh



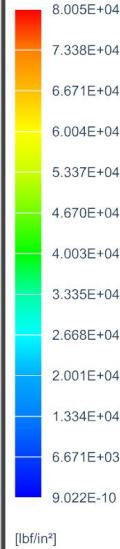
BCs + Loads

- A36 Steel
- 310 Stainless steel
- Inconel 718 aged

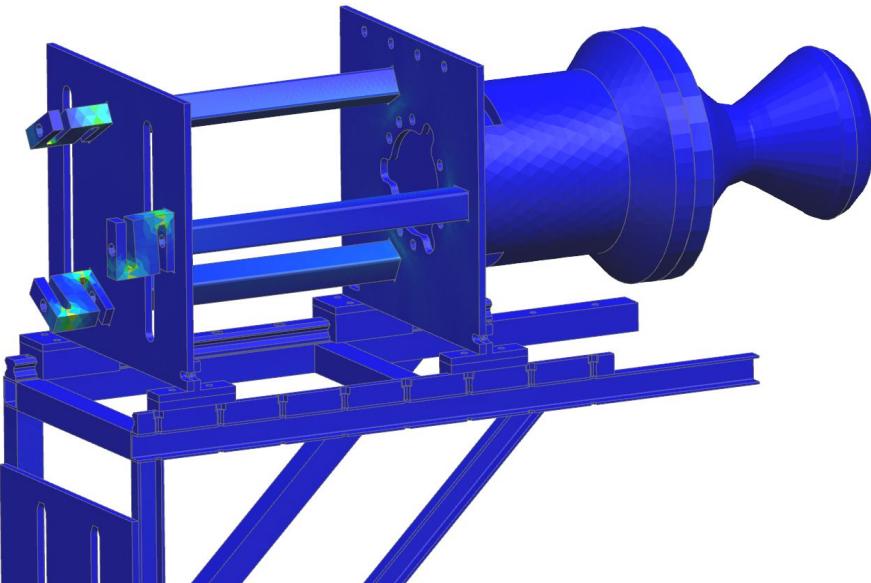
- ⊗ Fixed BC
- ⊗— Simply Supported BC
- Thrust:  $2250 \text{ lbf} \times 2 = 4500 \text{ lbf}$
- ◆ Glue contact

# Structural Analysis - FEA

ATLAS ENGINE MOUNT ASSEMBLY SIM 1 : Solution 1 Result  
Subcase - Statics 1, Static Step 1  
Stress - Elemental, Von-Mises  
Min : 9.022E-10, Max : 8.005E+04, Units = lbf/in<sup>2</sup>  
Deformation : Displacement - Nodal Magnitude



x10 Absolute Deformation



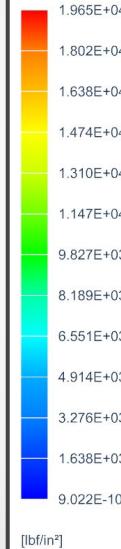
VMSTRESS 1

~ 14000 psi

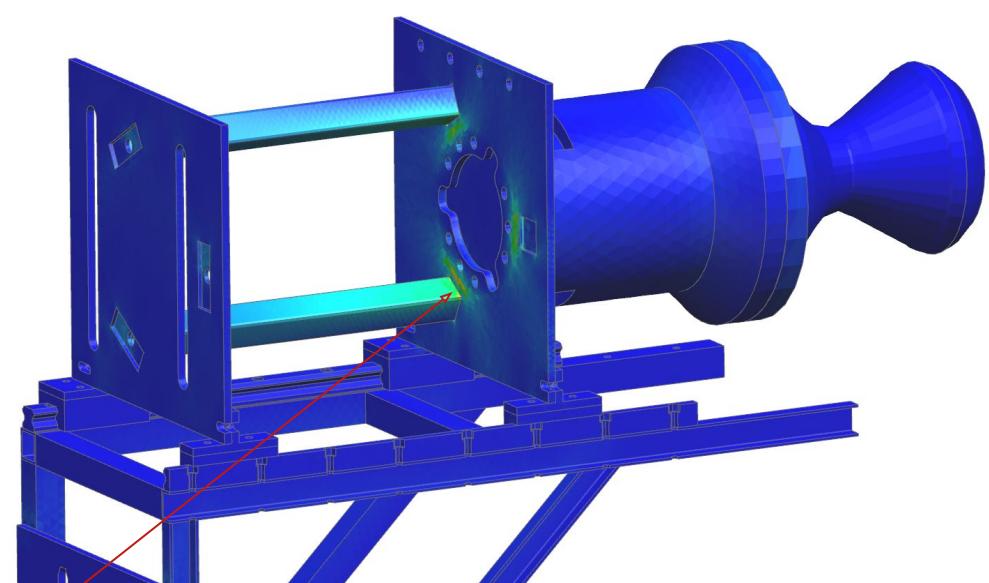
$$\sigma_{y, A36} = 36,300 \text{ psi}$$

$$SF_{\min} \approx 2.6$$

ATLAS ENGINE MOUNT ASSEMBLY SIM 1 : Solution 1 Result  
Subcase - Statics 1, Static Step 1  
Stress - Elemental, Von-Mises  
Min : 9.022E-10, Max : 8.005E+04, Units = lbf/in<sup>2</sup>  
Deformation : Displacement - Nodal Magnitude



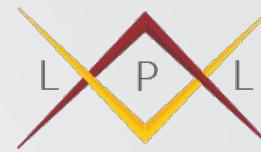
x10 Absolute Deformation



VMSTRESS 2

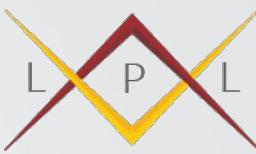
\*contour rescaled after hiding  
load cells

# Material Costs



Part	Description	Individual Cost [\$]	Quantity	Total Cost [\$]	Part Number	Manufacturer
Steel Square Cross Sectional Tubing	1" x 0.12" Carbon Steel Square Tube A500/A513 Hot Rolled, 6ft	32.95	2	65.9	10305	Online Metals
Steel Guide Rails	20 mm Wide Guide Rail for Ball Bearing Carriage, Rail Length 280mm	109	2	218	6709K63	McMaster
Steel Carriages	Ball Bearing Carriage for 20 mm Wide Rail	127	4	508	6709K18	McMaster
Engine Mount Plate	1/4 inch THICK A36 Steel Plate 1ftx4ft	96.76	1	96.76	P112	Metals Depot
Steel Rectangular Bar Stock	1in x 2in. Hot Rolled A-36 Steel Rectangle 4ft	51.96	1	51.96	P1112	Metals Depot
S-Beam Load Cell	High Accuracy, Stainless Steel, S-Beam Load Cell	122.1	3	366.3	LC103B-1K	ATO
M8 hex nut	x50 Zinc Yellow-Chromate Plated Steel Hex NutMedium-Strength, Class 8, M8 x 1.25 mm Thread	5.93	1	5.93	97700A160	McMaster
M8 25mm Steel Socket Head Screw	x50 Alloy Steel Socket Head ScrewBlack-Oxide, M8 x 1.25 mm Thread, 25 mm Long	12.67	1	12.67	91290A432	McMaster
M8 1mm Steel Socket Head Screw	x10 Fine-Thread Alloy Steel Socket Head Screw M8 x 1 mm Thread, 16 mm Long	8.33	1	8.33	96144A216	McMaster
1/2"-20 UNF Flat Head Screw	x5 Zinc-Plated Alloy Steel Hex Drive Flat Head Screw1/2"-20 Thread Size, 1" Long	7.41	2	14.82	91263A795	McMaster
5/8"-11 UNC Steel Hex Bolt	x5 High-Strength Grade 8 Steel Hex Head ScrewZinc-Aluminum Coated, 5/8"-11 Thread Size, 1-1/2" L	9.39	1	9.39	91286A397	McMaster
5/8" Steel Hex Nut	x25 Medium-Strength Steel Hex NutGrade 5, Zinc-Plated, 5/8"-11 Thread Size	14.26	1	14.26	95462A533	McMaster
5/8" Steel Washer	x10 316 Stainless Steel Washerfor 5/8" Screw Size, 0.688" ID, 1.5" OD	8.2	1	8.2	90107A035	McMaster
1/2"-20 UNF Threaded Stud	x1 Grade B7 Medium-Strength Steel Threaded Rod1/2"-20 Thread Size, 1" Longx	2.41	4	9.64	98750A478	McMaster
				1390.16		

# Load Cells



## OMEGA

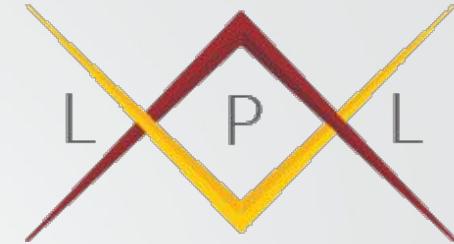
- Specified Load: 2000 lbf (8900 N)
- Safe Overload: 150%
- Width: 27.94 mm
- Length: 50.8 mm
- Input Impedance: 430 +/- 50 Ohms
- Output Impedance: 351 +/- 2 Ohms
- Operating Temp: -35 to 65 Degrees Celsius
- Price: **\$282.49**



## ATO

- Specified Load: 2200 lbf (9800 N)
- Safe Overload: 150%
- Width: 25.4 mm
- Length: 51 mm
- Input Impedance: 380 +/- 10 Ohms
- Output Impedance: 350 +/- 3 Ohms
- Operating Temp: -20 to 65 Degrees Celsius
- Price: **\$122.10**





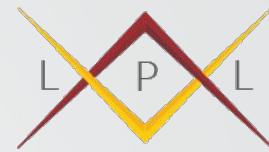
# Testing Campaign

## Liquid Propulsion Laboratory



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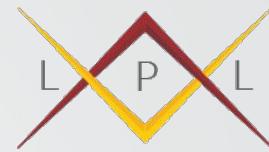
This will be done by subjecting the individual sections to 1.5x maximum expected operating pressure (MEOP), using a manual lever-pump (water is used instead of a compressible fluid as a safety precaution). This type of testing screens for acceptable workmanship and manufacturing defects.

## **Proof pressure:**

2200 psi for downstream of regulator pressurant lines, main lines and tanks

4050 psi for upstream of regulator pressurant lines

\*Ox Side lines have to be Ox cleaned after proof testing



This leak test is designed to identify potential leaks from the fittings of the systems. The test will use Snoop Liquid Leak Detector to identify leaks in the fittings and lines, applied at intervals of 2, 4 and 5 minutes after initiating the test at a certain pressure. The test will look for the presence of leaks at 50 psi, 100 psi, 200 psi and 375 psi.

**Class I** - Small, uniform (approximately 1/16 inch diameter) long persisting bubbles.

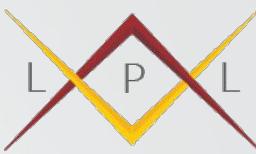
**Class II** - Mixture of random size bubbles, moderately persistent.

**Class III** - Large, fast forming bubbles of short persistence, most break as next one starts.

# Testing Campaign



- Qualifying the Test Stand
  - Hydrostatically proof test of all lines
  - Proof test of propellant tanks
  - Leak test of system
- Flow Tests
  - Fuel Side Cold Flow
  - LOx Side Cold Flow

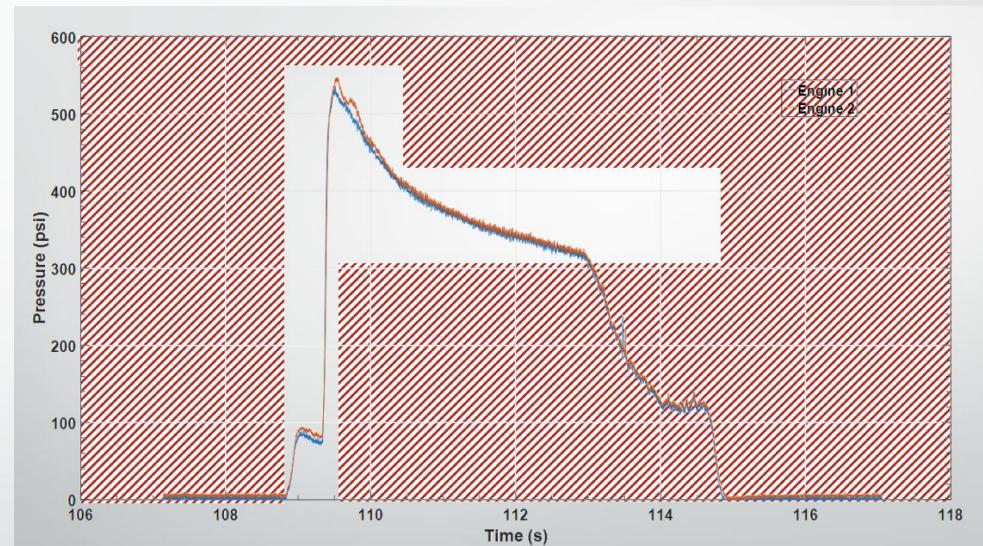


This will be done in order to characterize flow rate, droop, and pressure drop through the system. This is a high pressure test flow where we will emulate hot fire procedures. This test also serves to prepare TCs and TDs for actual hot fire.

Data from Cold Flows will be used to create abort gates.

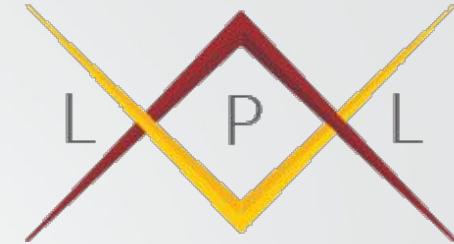
**Water** will be used for Fuel Side Cold Flow

**Cryogenic Nitrogen** will be used for Ox Side Cold Flow



We will machine an orifice with a specific Cv to simulate chamber pressure at the end of our system

\*Previously used WFTS to “tune” a Needle valve, however we learned from Hodor that that method was prone to errors due to issues with WFTS load cells



# Conclusions

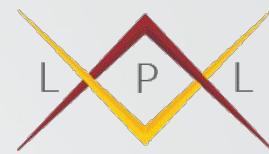
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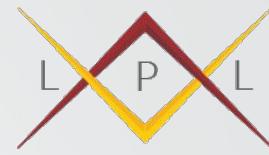


# Machining Costs



Part	Machining Needed	Cost
Tank Support Structure	Water Jetting, Welding	~\$150/hr
Engine Mount	Water Jetting, Welding	~\$150/hr
Orifice	Water Jetting	\$0
Tanks	Heat Treating, hole tapping	~\$400

# Overall Cost Breakdown



<b>Feed System Cost</b>	\$32,521.89
<b>Tank Support Structure Cost</b>	\$408.26
<b>Machining Costs</b>	\$2000
<b>Mount Structure Cost</b>	\$1390
<b>Allocated DAQ Cost</b>	\$5000
<b>Total Cost w/ 15% margin</b>	\$47,518.17

# Question/Comments?

