## ESERCIZI GEOMETRIA dulle AREE

Ripesso

$$\{S\} = \left\{ \begin{array}{l} Sy \\ Sx \end{array} \right\} = \left\{ \begin{array}{l} \int_{A} x \, dA \\ A \end{array} \right\} \quad \text{Se e'origine $\vec{e}$ G (BARICENTRO)} \qquad \text{or $\vec{e}$} = \frac{Sy}{A}$$

$$= \sum_{A} \int_{A} y \, dA \quad \text{or } A = 0$$

$$= \sum_{A} \sum_{A} \left\{ \left[ \int_{A} x \, dA \right] \right\} = \sum_{A} \sum_{A} \left[ \int_{A} x \, dA \right] = \sum_{A} \sum_{A} \left[ \int_{A} x$$

$$[I] = \begin{bmatrix} I_y & I_{xy} \\ I_{xy} & I_x \end{bmatrix} = \begin{bmatrix} \int x^2 dA & \int xy dA \end{bmatrix}$$

Se sisteme di referemento PRINCIPALE (=> Izy=0 -> gai orri sono ASSI PRINCIPALI D'INERZIA

Se un sistema

di riferimenta e

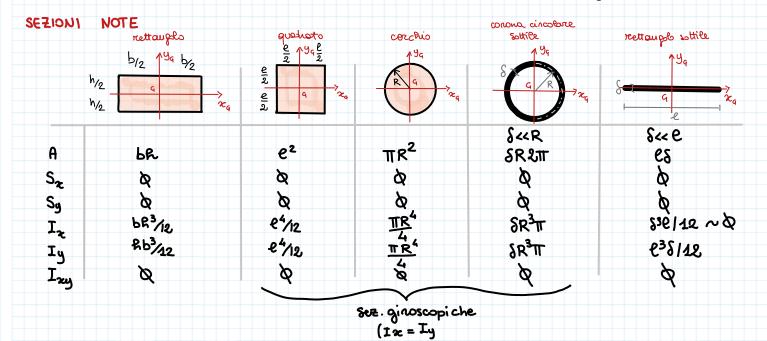
BARICENTRICO e

PRINCIPALE

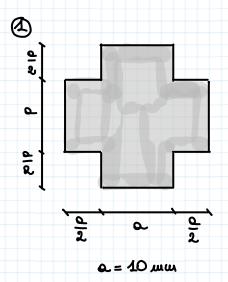
CENTRALE

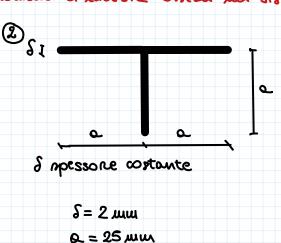
Sz, Sy = 0

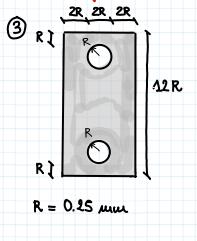
Izy = 0

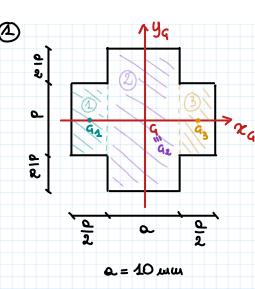


ESERCIZI - Calcolare i momenti d'inerzia ossieli nel sistema di riprimento controle









Si possono identificare due assi di simmetria assi de retta, quindi quegli essi sono anche assi principan e il Baniantro  $G \in A \times_a Ay_a$ .

Suddivido in 3 nettougoli & liqua:

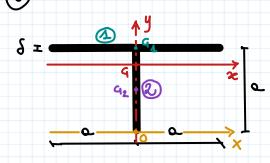
$$I_{2} = I_{24}^{4} + I_{24}^{2} + I_{24}^{3} =$$

$$= \frac{2 \cdot 0^{3}}{12} + \frac{0 \cdot (20)^{3}}{12} + \frac{2 \cdot 0^{3}}{12} = \frac{0^{4}}{24} + \frac{8}{12} \cdot 0^{4} + \frac{0^{4}}{24} = \frac{3}{4} \cdot 0^{4} = \frac{3}{4} \cdot (10)^{4} = 7500$$

$$12 = \frac{3}{4} \cdot 0^{3} + \frac{3}{4} \cdot 0^{4} = \frac{3}{4} \cdot 0^{4}$$

i termini di trasporto sono mulli perche i banicuitri Romo trutti le stesse ordinate

In maniera analogo secolo Iy:
$$Ty = \frac{4}{2} + \frac{3}{4} +$$



le rezione satile riportate presente un esse di rym essiale retto, quindi e auche erre principale d'invezione el partierzo un rist di rif. di parteuza centrato in 0 e uso le formule del trasporto del momento statica per ricavare la posizione del basicentra (y<sub>6</sub>).

considero la rezione formata de 2 rettangoli sottili:

$$S_{\infty} = S_{\infty}^{(1)} + S_{\infty}^{(2)} = 20.5 \cdot 0. + 0.5 \cdot \frac{0}{2} = \frac{5}{2} 0.5$$

NB se avezi scelto l'origine =  $G_{\infty}$ 

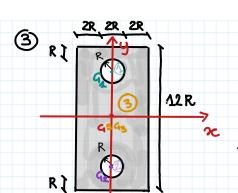
il corpo (1) avezbre avuto  $S_{\infty}^{(1)} = 0.$ 

A = 2a6 + a6 = 3a6  $y_{q} = \frac{5}{2}a^{2}6/3a6 = \frac{5}{6}a \quad \text{TRASIO IL SIST de RIF in } Q(0,\frac{5}{6}a) \quad \text{Questo et all sixt de air.} \quad \text{CENTRALE}$ 

Calcolo i momenti d'inertia 
$$I_{x_q} = I_{y_q}$$
:
$$I_{x_q} = I_{x_q}^{2} + I_{x_q}^{2} = \frac{245^3}{12} + \left(\frac{2}{6}\right)^2 265 + \frac{235}{12} + \left(\frac{5}{6}a - \frac{2}{2}\right)^2 a5 = \frac{35}{48} + \frac{3}{12} + \frac{3}{9}a^3 = \frac{235}{4} = \frac{(25)^3 \cdot 2}{4} = 7842,5 \text{ mu}^4$$

$$I_{y_q} = I_{y_q}^{2} + I_{y_q}^{2} = \frac{(2a)^3 \cdot 5}{12} + \frac{35}{12} = \frac{2}{3}a^3 \cdot 5 = \frac{2}{3}(25)^3 \cdot 2 = 20833,3 \text{ mu}^4$$

le rezione nivelte avere molte più inerzia attonno all'orre y



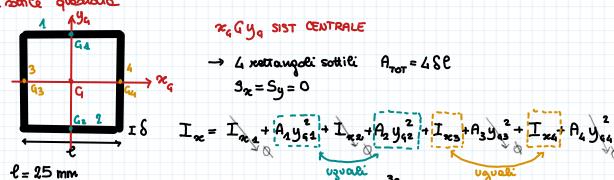
R = 0.25 mm

$$I_{2a} = \frac{GR \cdot (AZ)^{3}R^{3}}{12} - 2\left(\frac{\pi R^{4}}{4} + (4R)^{2}\pi R^{2}\right) = 864R^{4} - 2\left(\frac{65}{2}\pi R^{4}\right) = 864(0.25)^{4} - \frac{65}{2}\pi(0.25)^{4} = 2.976 \text{ mm}^{4}$$

$$I_{3a} = \frac{6^{3}R^{3} \cdot 12R}{12} - 2\left(\frac{\pi R^{4}}{4}\right) = 216R^{4} - \frac{\pi R^{4}}{2} = 0.838 \text{ mm}^{4}$$

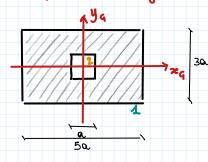
## ESERCIZI VISTI IN CLASSE ( MOU COMMENTATI)





 $\delta = 2 \, \text{mm} \qquad I_{\mathcal{R}} = 2 \, H_{1} \, y_{44}^{2} + 2 \, I_{33} = 2 \cdot S \, \ell \, \left(\frac{\varrho}{2}\right)^{2} + 2 \, \frac{\ell^{3} \delta}{12} = \frac{5 \, \ell^{3}}{2} + \frac{5 \, \ell^{3}}{6} = \frac{3}{3} \, S \, \ell^{3} = 20833.3 \, \text{mm}^{4}$   $I_{\mathcal{Y}} = \text{anabogo a } I_{\mathcal{X}} = 3 \, I_{y4} + 2 \, H_{3} \, \chi_{43}^{2} = 2 \, \frac{\ell^{3} \delta}{12} + 28 \, \ell \, \left(\frac{\ell}{2}\right)^{2} = \frac{\ell^{3} \delta}{6} + \frac{\ell^{3} \delta}{2} = \frac{2}{3} \, \delta \, \ell^{3} \qquad \text{Chroscoffical}$ 

## · Set. piena rettanoplare con foro quadrato



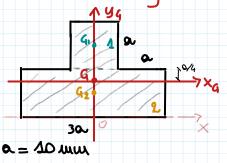
$$A_{TOT} = 3a \cdot 5a - a^{2} = 1/4a^{2}$$

$$I_{\infty} = \frac{b_{1}h_{1}^{3}}{12} - \frac{b_{2}R_{2}^{3}}{12} = \frac{5a(3a)^{3}}{12} - \frac{a^{4}}{12} = \left(\frac{135}{12} - \frac{1}{12}\right)a^{4} = \frac{67}{6}a^{4}$$

$$I_{y} = \frac{b_{1}^{3}h_{1}}{12} - \frac{b_{2}^{3}h_{2}}{12} = \frac{(5a)^{3}3a}{12} - \frac{a^{4}}{42} = \left(\frac{375}{12} - \frac{1}{12}\right)a^{4} = \frac{187}{6}a^{4}$$

$$Con a = 5mm \quad A = 200 \text{ mm}^{2}, \quad I_{\infty} = 6979, 2mm^{4}, \quad I_{y} = 19479, 2mm^{4}$$

## · Set. 1 asse di sym



$$A = A_{A} + A_{2} = \alpha^{2} + 3\alpha^{2} = 4\alpha^{2} = 4(3\alpha)^{2} = 40^{2}$$

$$S_{x} = 3\alpha^{2} \cdot \frac{\alpha}{2} + \alpha^{2} \cdot \frac{3}{2}\alpha = \frac{3}{2}\alpha^{3} + \frac{3}{2}\alpha^{3} = 3\alpha^{3}$$

$$I_{x} = \frac{\alpha^{4}}{12} + \alpha^{2}(\frac{\alpha}{2} + \frac{\alpha}{4})^{2} + \frac{3\alpha \cdot \alpha}{12} + 3\alpha^{2}(\frac{\alpha}{2})^{2} = \frac{\alpha^{4}}{12} + \frac{3}{16}\alpha^{4} = \frac{4}{4}\alpha^{2} + \frac{3}{46}\alpha^{4} = \frac{4}{48}\alpha^{4} = \frac{43}{42}\alpha^{4}$$

$$I_{y} = \frac{\alpha^{4}}{12} + \alpha^{2}(\alpha)^{2} + \frac{3\alpha^{3}\alpha}{12} + 3\alpha^{2}(\alpha) = \frac{\alpha^{4}}{12} + \frac{27\alpha^{4}}{12} = \frac{3}{3}\alpha^{4}$$

$$I_{x} = 1,083 \cdot 10^{4} \text{ mm}^{4}$$

$$I_{y} = 2,333 \cdot 10^{4} \text{ mm}^{4}$$